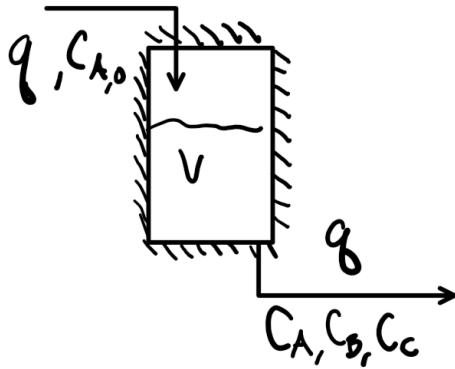


1)

System: Tank with contents and valves

- $A \xrightarrow{k_1} B, A \xrightarrow{k_2} C$
- $r_A = -(k_1 + k_2)C_A, r_B = k_1C_A, r_C = k_2C_A$
- $\rho, V, q = \text{const.}$
- Perfect mixing, Adiabatic, Isothermal
- $C_{B,0} = C_{C,0} = 0$

Variables:

Inputs	Units
$C_{A,0}$	kmol/m^3
Outputs	Units
C_A, C_B, C_C	mol/m^3

Parameters (Constant):

Var.	Units
q	m^3/min
V	m^3
k_1, k_2	min^{-1}

Component Balances

Since V is constant,

$$V \frac{dC_A}{dt} = q(C_{A,0} - C_A) + r_A V,$$

Using $r_A = -(k_1 + k_2)C_A$ and collecting the C_A terms,

$$V \frac{dC_A}{dt} + [q + (k_1 + k_2)V] C_A = qC_{A,0}.$$

Isolating for C_A ,

$$\left[\frac{V}{q + (k_1 + k_2)V} \right] \frac{dC_A}{dt} + C_A = \left[\frac{q}{q + (k_1 + k_2)V} \right] C_{A,0}.$$

Let,

$$\boxed{\tau_A = \frac{V}{q + (k_1 + k_2)V}, \quad K_A = \frac{q}{q + (k_1 + k_2)V}},$$

then,

$$\boxed{\tau_A \frac{dC_A}{dt} + C_A = K_A C_{A,0}}.$$

The B balance follows similarly, but with $C_{B,0} = 0$,

$$V \frac{dC_B}{dt} = -qC_B + k_1 C_A V,$$

$$\Rightarrow \left[\frac{V}{q} \right] \frac{dC_B}{dt} + C_B = \left[\frac{k_1 V}{q} \right] C_A.$$

Let,

$$\tau_B = \frac{V}{q}, \quad K_B = \frac{k_1 V}{q},$$

then,

$$\tau_B \frac{dC_B}{dt} + C_B = K_B C_A.$$

By analogy, C is nearly the same as B,

$$V \frac{dC_C}{dt} = -qC_C + k_2 C_A V,$$

$$\Rightarrow \left[\frac{V}{q} \right] \frac{dC_C}{dt} + C_C = \left[\frac{k_2 V}{q} \right] C_A.$$

Let,

$$\tau_C = \frac{V}{q}, \quad K_C = \frac{k_2 V}{q}.$$

then,

$$\tau_C \frac{dC_C}{dt} + C_C = K_C C_A.$$

Transfer Functions

Define steady state and deviation variables. The steady state variables are found by setting the time derivatives to 0,

$$\bar{C}_A = K_A \bar{C}_{A,0}, \quad \bar{C}_B = K_B \bar{C}_A, \quad \bar{C}_C = K_C \bar{C}_A.$$

Therefore, subtracting the steady state variables from the balance equations gives the deviation form,

$$\tau_A \frac{dC'_A}{dt} + C'_A = K_A C'_{A,0}, \quad \tau_B \frac{dC'_B}{dt} + C'_B = K_B C'_A, \quad \tau_C \frac{dC'_C}{dt} + C'_C = K_C C'_A,$$

where,

$$C'_i = C_i - \bar{C}_i.$$

Taking the Laplace transform of each balance equation,

$$C'_A(s) [\tau_A s + 1] = K_A C'_{A,0}(s),$$

$$C'_B(s) [\tau_B s + 1] = K_B C'_A(s),$$

$$C'_C(s) [\tau_C s + 1] = K_C C'_A(s).$$

Rearranging,

$$\begin{aligned}
 C'_A(s) &= \left[\frac{K_A}{\tau_A s + 1} \right] C'_{A,0}(s), \\
 C'_B(s) &= \left[\frac{K_B}{\tau_B s + 1} \right] C'_A(s) = \left[\frac{K_B}{\tau_B s + 1} \right] \left[\frac{K_A}{\tau_A s + 1} \right] C'_{A,0}(s), \\
 C'_C(s) &= \left[\frac{K_C}{\tau_C s + 1} \right] C'_A(s) = \left[\frac{K_C}{\tau_C s + 1} \right] \left[\frac{K_A}{\tau_A s + 1} \right] C'_{A,0}(s).
 \end{aligned}$$

Thus, the transfer functions are,

$$\begin{aligned}
 C'_A(s) \leftrightarrow C'_{A,0}(s) : \quad G_A(s) &= \frac{K_A}{\tau_A s + 1} \\
 C'_B(s) \leftrightarrow C'_{A,0}(s) : \quad G_B(s) &= \left[\frac{K_B}{\tau_B s + 1} \right] \left[\frac{K_A}{\tau_A s + 1} \right] \\
 C'_C(s) \leftrightarrow C'_{A,0}(s) : \quad G_C(s) &= \left[\frac{K_C}{\tau_C s + 1} \right] \left[\frac{K_A}{\tau_A s + 1} \right]
 \end{aligned}$$

The numerical values of the time constants and gains are given in the following table.

i	τ_i (min)	K_i (·)
A	4	0.4
B	10	1
C	10	0.5

2)

a.

$$V \frac{dP}{dt} = C_0 P_0 - C_1 P_1 (P - P_1)$$

Let,

$$f(P_0, P_1, P) = C_0 P_0 - C_1 P_1 (P - P_1).$$

By a Taylor series expansion about the steady state,

$$\begin{aligned}
 f' &= \left(\frac{\partial f}{\partial P_0} \right)_{ss} P'_0 + \left(\frac{\partial f}{\partial P_1} \right)_{ss} P'_1 + \left(\frac{\partial f}{\partial P} \right)_{ss} P' \\
 &= [C_0] P'_0 + [-C_1 \bar{P} + 2C_1 \bar{P}_1] P'_1 - [C_1 \bar{P}_1] P'.
 \end{aligned}$$

Plugging this back into the original model,

$$V \frac{dP'}{dt} + [C_1 \bar{P}_1] P' = [C_0] P'_0 + [-C_1 \bar{P} + 2C_1 \bar{P}_1] P'_1.$$

By rearrangement,

$$\left[\frac{V}{C_1 \bar{P}_1} \right] \frac{dP'}{dt} + P' = \left[\frac{C_0}{C_1 \bar{P}_1} \right] P'_0 + \left[\frac{-C_1 \bar{P} + 2C_1 \bar{P}_1}{C_1 \bar{P}_1} \right] P'_1.$$

Define,

$$\tau = \frac{V}{C_1 \bar{P}_1}, \quad K_0 = \frac{C_0}{C_1 \bar{P}_1}, \quad K_1 = \frac{-\bar{P} + 2\bar{P}_1}{\bar{P}_1}.$$

Therefore,

$$\tau \frac{dP'}{dt} + P' = K_0 P'_0 + K_1 P'_1.$$

Taking the Laplace transform,

$$P'(s) [\tau s + 1] = K_0 P'_0(s) + K_1 P'_1(s),$$

or by rearrangement,

$$P'(s) = \left[\frac{K_0}{\tau s + 1} \right] P'_0(s) + \left[\frac{K_1}{\tau s + 1} \right] P'_1(s).$$

Thus, the transfer functions are,

$$\begin{aligned} P'_0(s) \leftrightarrow P'(s) : \quad G_0(s) &= \frac{K_0}{\tau s + 1} \\ P'_1(s) \leftrightarrow P'(s) : \quad G_1(s) &= \frac{K_1}{\tau s + 1} \end{aligned}$$

The numerical values of the time constants and gains are given in the following table.

i	τ_i (min)	K_i (·)
0	2.083	0.1667
1	2.083	0.6667

b.

By the linear model from part a, the time derivative is set to 0,

$$P' = K_0 \bar{P}'_0 + K_1 \bar{P}'_1.$$

By the definition of deviation variables,

$$\begin{aligned} P &= \bar{P} + K_0 (P_0 - \bar{P}_0) + K_1 (P_1 - \bar{P}_1) \\ &= 40 \text{ psia} + (0.1667) (60 - 60) \text{ psia} + (0.6667) (20 - 30) \text{ psia} \\ &= 33.3 \text{ psia}. \end{aligned}$$

Therefore, the new steady state is $\boxed{P = 33.3 \text{ psia}}$. For the nonlinear model, take the original equation and set the time derivative to 0,

$$0 = C_0 \bar{P}_0 - C_1 \bar{P}_1 (\bar{P} - \bar{P}_1),$$

solving for \bar{P} ,

$$\bar{P} = \bar{P}_1 + \frac{C_0 \bar{P}_0}{C_1 \bar{P}_1}.$$

Numerically,

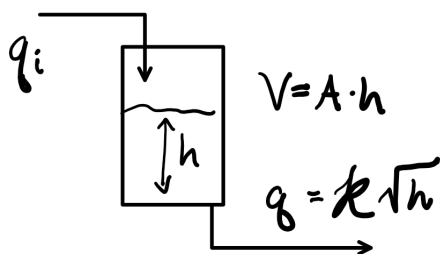
$$\boxed{\bar{P} = 20 \text{ psia} + \frac{\left(4 \frac{\text{ft}^3}{\text{min}}\right) (60 \text{ psia})}{\left(0.8 \frac{\text{ft}^3}{\text{min} \cdot \text{psia}}\right) (20 \text{ psia})} = 35 \text{ psia} .}$$

3)

System: Tank with contents and valves

• $A, \rho = \text{const.}$

• $q = k\sqrt{h}$



Variables:

Inputs	Units
q_i	ft^3/hr
Outputs	Units
h	ft

Parameters (Constant):

Var.	Units
A	ft^2
k	$\text{ft}^3/\text{hr}/\text{ft}^{1/2}$

a.

By a mass balance,

$$\frac{d(\rho V)}{dt} = \rho (q_i - q).$$

Using $\rho, A = \text{const.}$ and $q = k\sqrt{h}$,

$$\boxed{A \frac{dh}{dt} = q_i - k\sqrt{h}}.$$

b.

Let $f(q_i, h) = q_i - k\sqrt{h}$, then by a Taylor series expansion about the steady state,

$$\begin{aligned} f' &= \left(\frac{\partial f}{\partial q_i} \right)_{SS} q'_i + \left(\frac{\partial f}{\partial h} \right)_{SS} h' \\ &= [1] q'_i - \left[\frac{k}{2\bar{h}^{1/2}} \right] h'. \end{aligned}$$

Plugging this into the model,

$$A \frac{dh'}{dt} + \left[\frac{k}{2\bar{h}^{1/2}} \right] h' = q'_i$$

isolating for h' ,

$$\left[\frac{2A\bar{h}^{1/2}}{k} \right] \frac{dh'}{dt} + h' = \left[\frac{2\bar{h}^{1/2}}{k} \right] q'_i.$$

Let,

$$\boxed{\tau = \frac{2A\bar{h}^{1/2}}{k}, \quad K = \frac{2\bar{h}^{1/2}}{k}},$$

then,

$$\tau \frac{dh'}{dt} + h' = K q'_i.$$

Taking the Laplace transform and rearranging,

$$h'(s) = \left[\frac{K}{\tau s + 1} \right] q'_i(s).$$

Therefore, the transfer function of the linearized model is,

$$\boxed{h'(s) \leftrightarrow q'_i(s) : \quad G(s) = \frac{K}{\tau s + 1}}.$$

c.

Calculate the cross-sectional area of the tank, A ,

$$A = \frac{\pi D^2}{4} = \frac{\pi (36 \text{ ft}^2)}{4} = 28.27 \text{ ft}^2.$$

At steady state, $q = k\sqrt{h}$ must still hold, thus,

$$k = \frac{\bar{q}}{\sqrt{\bar{h}}} = \frac{16 \text{ ft}^3/\text{hr}}{(5 \text{ ft})^{1/2}} = 7.155 \text{ ft}^3/\text{hr}/\text{ft}^{1/2}.$$

By the definition of τ ,

$$\boxed{\tau = \frac{2A\bar{h}^{1/2}}{k} = \frac{2(28.27 \text{ ft}^2)(5 \text{ ft})^{1/2}}{7.155 \text{ ft}^3/\text{hr}/\text{ft}^{1/2}} = 17.67 \text{ hr}}.$$

By the definition of K ,

$$K = \frac{2\bar{h}^{1/2}}{k} = \frac{2(5 \text{ ft})^{1/2}}{7.155 \text{ ft}^3/\text{hr}/\text{ft}^{1/2}} = 0.625 \text{ hr}/\text{ft}^2.$$

d.

At the new steady state, the linearized model predicts,

$$h' = Kq'_i,$$

or,

$$h = \bar{h} + K(q_i - \bar{q}_i).$$

Numerically,

$$h = 5 \text{ ft} + (0.625 \text{ hr}/\text{ft}^2)(24 - 16) \text{ ft}^3/\text{hr} = 10 \text{ ft}.$$

Under the linearized model, there is no problem since the liquid level is exactly 10 ft, which is the height of the tank. However, this is a linearized model, and does not account for nonlinear effects. Using the original dynamic model, the steady state condition is given as,

$$\bar{h} = \left(\frac{\bar{q}_i}{k}\right)^2 = \left(\frac{16 \text{ ft}^3/\text{hr}}{7.155 \text{ ft}^3/\text{hr}/\text{ft}^{1/2}}\right)^2 = 11.25 \text{ ft}.$$

This is larger than the tank height, which is problematic.