CS 376 Hybrid Systems: HW 8

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19 November 2014

1 Hybrid Simulation of a Tank Sytem

Consider the two-tank systems shown in Fig. 1. The system consists of two identical cylindrical tanks, of unlimited height, that are connected by a pipe at level h. We denote by h_1 and h_2 the water levels in tanks 1 and 2 respectively. The input flow Q_{in} is provided by a pump and it is described by

$$Q_{in} = V_{in}k_{in}u(t), (1)$$

where $V_{in} \in \{0,1\}$ represents a valve that can be used to turn on or off the pump (no partially open valve), k_{in} is a linear gain, and u(t) is the input signal representing the flow at the pump. The flow Q_a between the two tanks is controlled by a valve V_a . An outlet value V_{out} located at the bottom of tank 2 is used to empty the tank. Tank 2 is equipped with a sensor that measures the output flow which is described by

$$Q_{out} = V_{out} k_{out} \sqrt{\rho g h_2} \tag{2}$$

where $V_{out} \in \{0, 1\}$ represents the outlet valve, k_{out} is a linear gain, ρ is the density of the water, and g is the gravitational constant.

The dynamic evolution of the systems is described by

$$\dot{h_1} = \frac{1}{A}(Q_{in} - Q_a) \tag{3}$$

$$\dot{h_2} = \frac{1}{A}(Q_a - Q_a) \tag{4}$$

where A is the section of the identical cylindrical tanks. Following Toricelli's law, the flow Q_a depends on the water levels h_1 and h_2 as follows:

$$Q_{a} = \begin{cases} 0, & \text{if } h_{1} < h \land h_{2} < h \\ V_{a}k_{a}\sqrt{\rho g(h_{1} - h)}, & \text{if } h_{1} > h \land h_{2} < h \\ V_{a}k_{a}\sqrt{\rho g(h - h_{2})}, & \text{if } h_{1} < h \land h_{2} > h \\ sign(h_{1} - h_{2})V_{a}k_{a}\sqrt{\rho g|h_{1} - h_{2}|}, & \text{if } h_{1} > h \land h_{2} > h \end{cases}$$

$$(5)$$

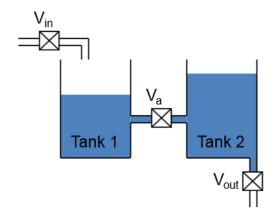


Figure 1: Two-tank system

where $V_a \in \{0,1\}$ and k_a is a linear gain. The evolution of the continuous state can be described by

$$x = [h_1, h_2]^T \tag{6}$$

$$\dot{x} = f_q(x(t), u(t)) \tag{7}$$

where q is the discrete mode of the sytem. The mode transitions are based on guard conditions indicated in Equation 5.

Assume the following values for the system parameters and initial conditions:

$$V_{in} = V_a = V_{out} = 1$$
, all valves are open (8)

(9)

u(t) is a pulse with amplitude 1 m^3 and frequency 1 Hz resulting in an average rate of 1 m^3/sec .

$$h = 0.3 \text{ m}$$
 (10)

$$k_{in} = 0.06,$$
 (11)

$$k_a = 0.001,$$
 (12)

$$k_{out} = 0.001,$$
 (13)

$$g = 9.81 \ m/sec^2 \ , \tag{14}$$

$$\rho = 1000 \ kg/m^3 \ , \tag{15}$$

$$A = 0.0154 \ m^2 \tag{16}$$

2 A hybrid automaton model

The formal model for the two-tank system hybrid automaton.

$$H = (q, X, Init, f, Inv, E, G, R)$$

$$\tag{17}$$

The set of discrete modes.

if
$$h_1 < h \land h_2 < h$$
, : separated (18)

if
$$h_1 > h \wedge h_2 < h$$
,: $from_1$ (19)

if
$$h_1 < h \land h_2 > h$$
,: $from_2$ (20)

if
$$h_1 > h \land h_2 > h$$
, : balancing (21)

$$q = \{separated, from_1, from_2, balancing\}$$
 (22)

The set of continuous variables. $X = \mathbb{R}$

$$X = \{u(t), x, Q_a, Q_{out}\}\tag{23}$$

The set of initial conditions. $Init \subseteq Q \times X$

$$Init = \{\} \tag{24}$$

The vector field. $f: Q \times X$

$$f = f_{separated} \cup f_{from_1} \cup f_{from_2} \cup f_{balancing}$$
 (25)

The invariant set. (all empty) $Q\mapsto 2^X$

$$Inv = Inv_{separated} \cup Inv_{from_1} \cup Inv_{from_2} \cup Inv_{balancing}$$
 (26)

The invariants are the same for each mode. $Inv_i =$

$$h_1 \ge 0 \tag{27}$$

$$h_2 \ge 0 \tag{28}$$

 $f_{separated} =$

$$Q_{in} = k_{in} \ m/sec^2 \tag{29}$$

$$Q_a = 0, (30)$$

$$\dot{h_1} = \frac{Q_{in}}{A} \tag{31}$$

$$\dot{h_2} = -\frac{Q_{out}}{A} \tag{32}$$

$$Q_o u t = k_{out} \sqrt{\rho g h_2} \tag{33}$$

 $f_{from_1} =$

$$Q_{in} = k_{in} \ m/sec^2 \tag{34}$$

$$Q_a = k_a \sqrt{\rho g(h_1 - h)},\tag{35}$$

$$\dot{h_1} = \frac{Q_{in} - Q_a}{A} \tag{36}$$

$$\dot{h_2} = \frac{Q_a - Q_{out}}{A} \tag{37}$$

$$Q_o ut = k_{out} \sqrt{\rho g h_2} \tag{38}$$

 $f_{from_2} =$

$$Q_{in} = k_{in} \ m/sec^2 \tag{39}$$

$$Q_a = k_a \sqrt{\rho g(h - h_2)},\tag{40}$$

$$\dot{h_1} = \frac{Q_{in} - Q_a}{A} \tag{41}$$

$$\dot{h_1} = \frac{Q_{in} - Q_a}{A} \tag{41}$$

$$\dot{h_2} = \frac{Q_a - Q_{out}}{A} \tag{42}$$

$$Q_o ut = k_{out} \sqrt{\rho g h_2} \tag{43}$$

 $f_{balancing} =$

$$Q_{in} = k_{in} \ m/sec^2 \tag{44}$$

$$Q_a = sign(h_1 - h_2)k_a \sqrt{\rho g|h_1 - h_2|}$$
(45)

$$\dot{h_1} = \frac{Q_{in} - Q_a}{A} \tag{46}$$

$$\dot{h_2} = \frac{Q_a - Q_{out}}{A} \tag{47}$$

$$Q_out = k_{out}\sqrt{\rho g h_2} \tag{48}$$

The collection of discrete transitions. $E \subset Q \times Q$

$$E = \{(separated, from_1)$$
(49)

$$(separated, from_2)$$
 (50)

$$(separated, balanced)$$
 (51)

$$(from_1, separated)$$
 (52)

$$(from_1, from_2) (53)$$

$$(from_1, balanced)$$
 (54)

$$(from_2, from_1) (55)$$

$$(from_2, separated)$$
 (56)

$$(from_2, balanced)$$
 (57)

$$(balanced, from_1) (58)$$

$$(balanced, from_2)$$
 (59)

$$(balanced, separated)$$
 (60)

The guards on the transitions. $G: E \mapsto 2^X$

$$E = \{ (separated, from_1), h_1 > h \land h_2 < h,$$
(61)

$$(separated, from_2), h_1 < h \land h_2 > h, \tag{62}$$

$$(separated, balanced), h_1 > h \land h_2 > h, \tag{63}$$

$$(from_1, separated), h_1 < h \land h_2 < h,$$
 (64)

$$(from_1, from_2), h_1 < h \land h_2 > h, \tag{65}$$

$$(from_1, balanced), h_1 > h \land h_2 > h, \tag{66}$$

$$(from_2, from_1), h_1 > h \land h_2 < h, \tag{67}$$

$$(from_2, separated), h_1 < h \land h_2 < h,$$
 (68)

$$(from_2, balanced), h_1 > h \land h_2 > h,$$
 (69)

$$(balanced, from_1), h_1 > h \land h_2 < h, \tag{70}$$

$$(balanced, from_2), h_1 < h \land h_2 > h, \tag{71}$$

$$(balanced, separated), h_1 < h \land h_2 < h, \} \tag{72}$$

The reset relation on the transitions. $R: E \times X \mapsto 2^X$

$$R = \emptyset \tag{73}$$

3 Simulink Simulation

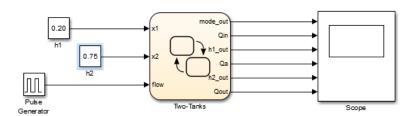


Figure 2: Two-tank SimuLink Model

The model is simulated for several initial conditions $x_0 = [h_1, h_2]^T$. The continuous state Q_a, x , discrete state q, and output Q_{out} of the system are plotted.

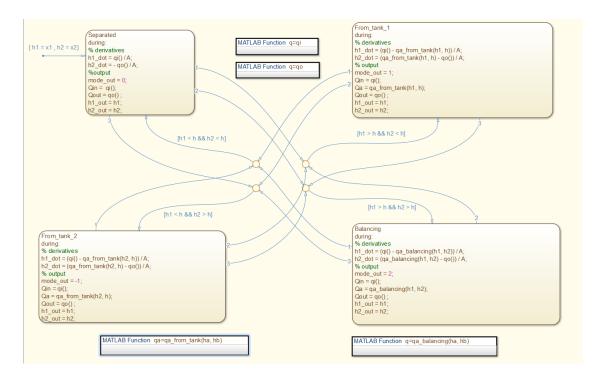


Figure 3: Two-tank StateFlow Model

```
| qa_from_tank × qo × qa_balancing × qi × + |
| function q=qi() |
| % QO Compute output flow rate from tank 2 |
| % A negative flow value should be avoided. |
| % It would be non-physical. |
| 5 - q = Kin * flow; |
```

Figure 4: Input Flow Function

Figure 5: Output Flow Function

```
3.1 x_0 = [0.20, 0.75]^T
```

3.2
$$x_0 = [0.50, 0.20]^T$$

3.3
$$x_0 = [0.50, 0.50]^T$$

3.4
$$x_0 = [0.20, 0.20]^T$$

Figure 6: From Tank Flow Function

Figure 7: Balancing Flow Function

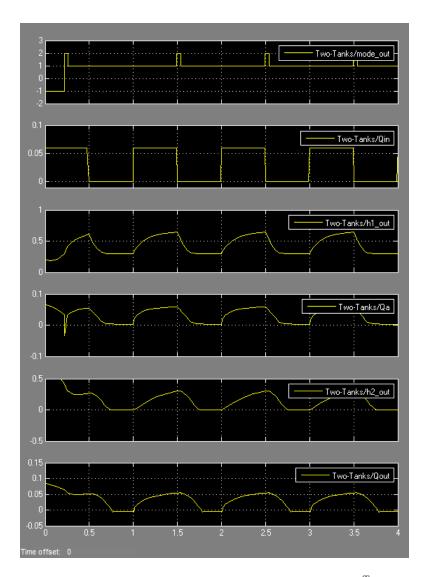


Figure 8: Two-tank SimuLink Model, $x_0 = [0.2, 0.75]^T$

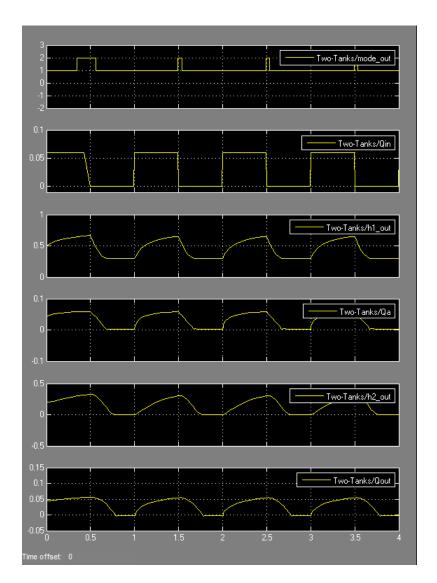


Figure 9: Two-tank SimuLink Model, $\boldsymbol{x}_0 = [0.50, 0.20]^T$

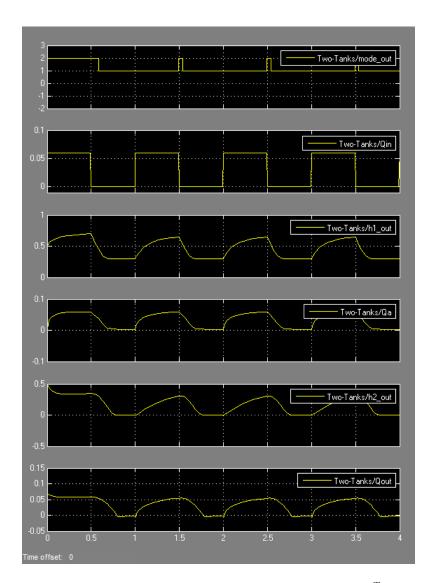


Figure 10: Two-tank SimuLink Model, $x_0 = [0.50, 0.50]^T$

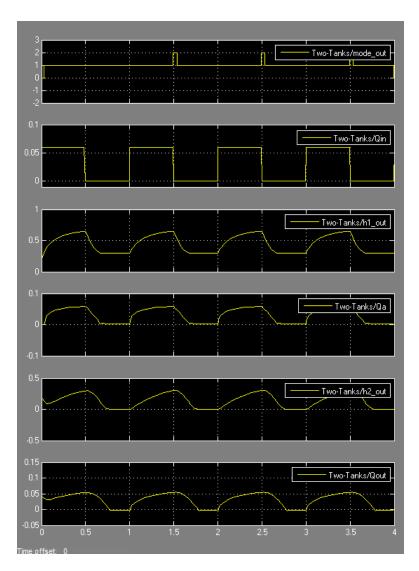


Figure 11: Two-tank SimuLink Model, $x_0 = [0.20, 0.20]^T$