Low Reynolds number gravitational settling of a sphere through a fluid-fluid interface: Modelling using a boundary integral method

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Abstract

1 Introduction

2 Theoretical Background

We present a theoretical construction of the boundary integral equations. Symbols used are defined in table 1.

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Table 1: Definition of symbols.

Symbol	Definition	
a	Sphere radius	
e	Exponential constant	
$\hat{e}_{lpha,ij}$	Greens function for strain in fluid α	
$f_{\mathrm{s},i}(oldsymbol{x}) = m_j(oldsymbol{x}) T_{1,ij}(oldsymbol{x})$	Traction on surface of sphere	
$f_{lpha,i}(oldsymbol{x}) = n_j(oldsymbol{x}) T_{lpha,ij}(oldsymbol{x})$	Traction of fluid α on interface	
\mathcal{F}_i	Arbitrary constant vector	
$oldsymbol{F}$	Force vector	
$\boldsymbol{g} = (-9.81 \mathrm{m \ s^{-2}}) \boldsymbol{\hat{z}}$	Acceleration due to gravity	
\mathcal{I}	Surface of interface	
$i = \sqrt{-1}$	The unit imaginary number	
J_{ij}	Components of tensor kernel for velocity Greens function	
K_{ijk}	Components of tensor kernel for stress Greens function	
k	Dummy variable in Fourier space	
$k = \mathbf{k} $	Magnitude of \boldsymbol{k}	
m	Outward normal to sphere surface	
n	Normal to interface (points into fluid 1)	
$P_{\alpha}(\boldsymbol{x})$	Pressure field of fluid α	
$P_{ ext{d},lpha}(oldsymbol{x})$	Dynamic pressure of fluid α	
\hat{P}_{lpha}	Greens function for dynamic pressure of fluid α	
$ar{P}_{lpha,i}(oldsymbol{\xi})$	Components of the vector of solutions for \hat{P}_{α} in equation 66	
$ ilde{P}_{lpha,i}(oldsymbol{k})$	Fourier transform of $\bar{P}_{\alpha,i}(\boldsymbol{\xi})$	
S	Arc length along interface measured from axis	
S	Surface of sphere	
$T_{lpha,ij}(oldsymbol{x})$	Stress tensor field of fluid α	
$\hat{T}_{lpha,ij}(oldsymbol{x'}-oldsymbol{y'})$	Stress Greens function for fluid α	
t	Time	
$oldsymbol{u}_{lpha}(oldsymbol{x})$	Velocity field of fluid α	
$oldsymbol{u}_{ ext{s}}=u_{ ext{s}}oldsymbol{\hat{z}}$	Velocity of sphere	

	$\hat{u}_{lpha,i}(oldsymbol{x'}-oldsymbol{y'})$,
	$ar{u}_{lpha,i,j}(oldsymbol{\xi})$	Components
	$ ilde{u}_{lpha,ij}(oldsymbol{k})$	
	\mathcal{V}_{lpha}	
x		
y		
\hat{z}		Unit
	$\alpha = 1, 2$	
δ_{ij}	$m{y}_i = egin{cases} 1 & ext{if } i = j \ 0 & ext{if } i eq j \end{cases}$ $m{y'}_i = egin{cases} 1 & ext{if } m{x} = m{y} \ 0 & ext{if } m{x} eq m{y} \end{cases}$	
$\delta(oldsymbol{x'} -$	y') = $\begin{cases} 1 & \text{if } x = y \\ 0 & \text{if } x \neq y \end{cases}$	
	η_lpha	
	heta	Pol
	$ ho_lpha$	
	$ ho_{ m s}$	
	σ	
ϕ		Azimu
$oldsymbol{\xi} = oldsymbol{x'} - oldsymbol{y'}$		
	$\xi = \boldsymbol{\xi} $	

Velocity Greens function for fluid α of the tensor of solutions for $\hat{u}_{\alpha,i}$ in equation 66 Fourier transform of $\bar{u}_{\alpha,i,j}(\boldsymbol{\xi})$ Volume of fluid α Position vector Position vector vector in the upward vertical direction Fluid label Kronecker delta Dirac delta function Viscosity of fluid α lar angle with respect to sphere centre Density of fluid α Sphere density Interfacial Tension uhtal angle with respect to axis of motion Dimensionless separation vector

Magnitude of $\boldsymbol{\xi}$

2.1 Equations of Motion

The starting point for all fluid dynamical problems are the continuity (equation 1) and Navier Stokes (equation 2) equations (Batchelor, 1967):

$$\frac{\partial \rho_{\alpha}(\boldsymbol{x},t)}{\partial t} + \boldsymbol{\nabla} \cdot [\rho_{\alpha}(\boldsymbol{x},t)\boldsymbol{u}_{\alpha}(\boldsymbol{x},t)] = 0, \tag{1}$$

$$\rho_{\alpha}(\boldsymbol{x},t) \left(\frac{\partial \boldsymbol{u}_{\alpha}(\boldsymbol{x},t)}{\partial t} + (\boldsymbol{u}_{\alpha}(\boldsymbol{x},t) \cdot \nabla) \boldsymbol{u}_{\alpha}(\boldsymbol{x},t) \right) =$$

$$-\nabla P_{\alpha}(\boldsymbol{x},t) - \rho_{\alpha}(\boldsymbol{x},t)g + \eta_{\alpha} \left[\nabla^{2} \boldsymbol{u}_{\alpha}(\boldsymbol{x},t) + \frac{\nabla(\nabla \cdot \boldsymbol{u}_{\alpha}(\boldsymbol{x},t))}{3} \right].$$
(2)

Forming a coupled set of non-linear, partial differential equations for the velocity and pressure fields, these represent mass and momentum conservation respectively and must be satisfied by all fluid phases within the system. For most practical applications, the fluids are assumed to be incompressible (have constant density) and so the continuity equation reduces to the incompressibility relation

$$\nabla \cdot \boldsymbol{u}_{\alpha}(\boldsymbol{x}, t) = 0. \tag{3}$$

This can be combined with equation 2 to form the incompressible Navier Stokes equation

$$\rho_{\alpha} \left(\frac{\partial \boldsymbol{u}_{\alpha}(\boldsymbol{x},t)}{\partial t} + (\boldsymbol{u}_{\alpha}(\boldsymbol{x},t) \cdot \boldsymbol{\nabla}) \boldsymbol{u}_{\alpha}(\boldsymbol{x},t) \right) = -\boldsymbol{\nabla} P_{\alpha}(\boldsymbol{x},t) - \rho_{\alpha} g + \eta_{\alpha} \nabla^{2} \boldsymbol{u}_{\alpha}(\boldsymbol{x},t).$$
(4)

The equations of motion can be expressed in an alternative form by defining the stress tensor $T_{\alpha}(\boldsymbol{x},t)$ (Manga, 1994) and dynamic pressure $P_{\mathrm{d},\alpha}(\boldsymbol{x},t)$:

$$\boldsymbol{T}_{\alpha}(\boldsymbol{x},t) = -P_{\mathrm{d},\alpha}(\boldsymbol{x},t)\boldsymbol{I} + \eta_{\alpha}[\boldsymbol{\nabla}\boldsymbol{u}_{\alpha}(\boldsymbol{x},t) + (\boldsymbol{\nabla}\boldsymbol{u}_{\alpha}(\boldsymbol{x},t))^{T}], \tag{5}$$

$$P_{d,\alpha}(\boldsymbol{x},t) = P_{\alpha}(\boldsymbol{x},t) - \rho_{\alpha}\boldsymbol{g} \cdot \boldsymbol{x}. \tag{6}$$

This definition of the stress tensor removes the gravitational body force from the equations of motion, meaning that it only appears in the boundary conditions. The Navier Stokes equation then becomes

$$\rho_{\alpha} \left(\frac{\partial \boldsymbol{u}_{\alpha}(\boldsymbol{x},t)}{\partial t} + (\boldsymbol{u}_{\alpha}(\boldsymbol{x},t) \cdot \boldsymbol{\nabla}) \boldsymbol{u}_{\alpha}(\boldsymbol{x},t) \right) = \boldsymbol{\nabla} \cdot \boldsymbol{T}_{\alpha}(\boldsymbol{x},t). \tag{7}$$

When working in fluid dynamics, it is usual to non-dimensionalise the equations of motion and boundary conditions (White, 1999). This can be achieved by scaling the quantities involved by parameters specific to the problem. For example, consider a problem with typical scales of length L_c and velocity U_c . This allows us to define dimensionless variables (denoted by a ')

$$\boldsymbol{x} = L_{c} \boldsymbol{x'}, \tag{8}$$

$$\boldsymbol{u}_{\alpha}(\boldsymbol{x},t) = U_{c}\boldsymbol{u'}_{\alpha}(\boldsymbol{x'},t'), \tag{9}$$

and

$$t = \frac{L_{\rm c}t'}{U_{\rm c}} \tag{10}$$

In the case of highly viscous flows, the pressure is dominated by viscosity and so the relevant scaling for the dynamic pressure uses a characteristic viscosity η_c and is given by Lee and Leal (1982)

$$P_{d,\alpha}(\boldsymbol{x},t) = \frac{\eta_c U_c P'_{d,\alpha}(\boldsymbol{x'},t')}{L_c}.$$
(11)

This choice of pressure scaling means that upon substitution of equations 8 to 11 into equation 5 the stress tensor can also be non-dimensionalised,

$$\boldsymbol{T}_{\alpha}(\boldsymbol{x},t) = \frac{\eta_{c}U_{c}\boldsymbol{T'}_{\alpha}(\boldsymbol{x'},t')}{L_{c}} \quad \text{where} \quad \boldsymbol{T'}_{\alpha}(\boldsymbol{x'},t') = p'_{d,\alpha}(\boldsymbol{x'},t')\boldsymbol{I} + \frac{\eta_{\alpha}[\boldsymbol{\nabla'}\boldsymbol{u'}_{\alpha}(\boldsymbol{x'},t') + (\boldsymbol{\nabla'}\boldsymbol{u'}_{\alpha}(\boldsymbol{x'},t'))^{T}]}{\eta_{c}}.$$
(12)

In this case, the continuity and Navier Stokes equations become

$$\nabla' \cdot \boldsymbol{u'}_{\alpha}(\boldsymbol{x'}, t') = 0 \tag{13}$$

and

$$\operatorname{Re}\left(\frac{\partial \boldsymbol{u'}_{\alpha}(\boldsymbol{x'},t')}{\partial t'} + (\boldsymbol{u'}_{\alpha}(\boldsymbol{x'},t') \cdot \boldsymbol{\nabla}')\boldsymbol{u'}_{\alpha}(\boldsymbol{x'},t')\right) = \boldsymbol{\nabla}' \cdot \boldsymbol{T'}_{\alpha}(\boldsymbol{x'},t'), \tag{14}$$

where the Reynolds number is defined as

$$Re_{\alpha} = \frac{\rho_{\alpha} L_{c} U_{c}}{\eta_{c}} \tag{15}$$

As we are considering the case of low Reynolds number ($\text{Re}_{\alpha} \ll 1$), we can neglect the inertial terms on the right hand side and the equation reduces to the Stokes equation Kim and Karrila (2005)

$$\nabla' \cdot T'_{\alpha}(x', t') = 0. \tag{16}$$

2.2 Boundary Conditions

In order to complete the formulation of any fluid dynamics problem, it is necessary to state the boundary conditions alongside the equations of motion (Riley et al., 2006). For fluids of infinite (or semi-infinite) extent in some dimension, these include the value of the flow velocity at infinity. For bounded flows, the conditions are imposed at the boundaries of the fluid domain, and their exact nature depends on the phase of the material bounding it.

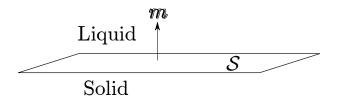


Figure 1: Definition of m and S for a fluid-solid boundary.

2.2.1 Fluid-Solid Boundary

At low Reynolds number for a fluid-solid boundary given by the surface S (see figure 1), the kinematic boundary condition (that on the velocity) states that both the normal and tangential components (with respect to the boundary) of velocity are the same as that of the solid at the boundary. This is easily expressed in dimensionless form as

$$u'_{\alpha}(x') = U'_{s} \quad x \in \mathcal{S}.$$
 (17)

There also needs to be a dynamic boundary condition applied at the interface. If the solid exerts a force F onto the fluid then the condition states

$$\int \boldsymbol{m} \cdot \boldsymbol{T}_{\alpha} d\mathcal{S} = \boldsymbol{F}. \tag{18}$$

Using the non-dimensionalisation scheme presented above this becomes

$$\eta_{\rm c} U_{\rm c} L_{\rm c} \int \boldsymbol{m} \cdot \boldsymbol{T'}_{\alpha} \mathrm{d} \mathcal{S'} = \boldsymbol{F}.$$
(19)

2.2.2 Fluid-Fluid Boundary

In the case of a fluid-fluid boundary (see figure 2), both a kinematic and dynamic boundary condition need to be satisfied. The kinematic boundary condition states that the velocity of the two fluids must be continuous across the interface (Manga, 1994). Expressed in



Figure 2: Definition of n and \mathcal{I} for a fluid-fluid boundary.

dimensionless form this looks like

$$\boldsymbol{u'}_{1}(\boldsymbol{x'}) = \boldsymbol{u'}_{2}(\boldsymbol{x'}), \quad \boldsymbol{x'} \in \mathcal{I}.$$
 (20)

The dynamic boundary condition is an expression of the balance between the stress discontinuity across the interface and the interfacial tension (IFT)(Manga, 1994):

$$\boldsymbol{n} \cdot [\boldsymbol{T}_1(\boldsymbol{x}) - \rho_1(\boldsymbol{g} \cdot \boldsymbol{x})\boldsymbol{I}] - \boldsymbol{n} \cdot [\boldsymbol{T}_2(\boldsymbol{x}) - \rho_2(\boldsymbol{g} \cdot \boldsymbol{x})\boldsymbol{I}] = \sigma(\boldsymbol{x})\boldsymbol{n}(\boldsymbol{\nabla}_{\mathrm{s}} \cdot \boldsymbol{n}) - \boldsymbol{\nabla}_{\mathrm{s}}\sigma(\boldsymbol{x}), \quad \boldsymbol{x} \in \mathcal{I}. \quad (21)$$

The operator ∇_s is defined as the tangential gradient operator within the surface \mathcal{I} :

$$\nabla_{s} = (I - nn) \cdot \nabla. \tag{22}$$

When this takes the normal vector as its argument it can be shown that (Brackbill et al., 1992)

$$\nabla_{\mathbf{s}} \cdot \boldsymbol{n} = \nabla \cdot \boldsymbol{n}. \tag{23}$$

The presence of gradients in the interfacial tension can lead to so-called Marangoni effects (Kim and Karrila, 2005). However, for present purposes we will assume that the interfacial tension is uniform across the interface S and so the last term on the right hand side equals

zero;

$$\boldsymbol{n} \cdot [\boldsymbol{T}_1(\boldsymbol{x}) - \rho_1(\boldsymbol{g} \cdot \boldsymbol{x})\boldsymbol{I}] - \boldsymbol{n} \cdot [\boldsymbol{T}_2(\boldsymbol{x}) - \rho_2(\boldsymbol{g} \cdot \boldsymbol{x})\boldsymbol{I}] = \sigma(\boldsymbol{x})\boldsymbol{n}(\boldsymbol{\nabla} \cdot \boldsymbol{n}), \quad \boldsymbol{x} \in \mathcal{S}.$$
 (24)

Like the equations of motion, this can be non-dimensionalised using equations 8 to 12:

Ca
$$\boldsymbol{n} \cdot (\boldsymbol{T'}_1 - \boldsymbol{T'}_2) + \operatorname{Bo}(\hat{\boldsymbol{z}} \cdot \boldsymbol{x'}) \boldsymbol{n} = (\boldsymbol{\nabla'} \cdot \boldsymbol{n}) \boldsymbol{n}.$$
 (25)

The capillary number Ca and Bond number Bo are dimensionless numbers defined as:

$$Ca = \frac{\eta_c U_c}{\sigma} \tag{26}$$

$$Bo = \frac{(\rho_2 - \rho_1)gL_c^2}{\sigma} \tag{27}$$

The $\nabla \cdot \mathbf{n}$ factor in equation 21 can be expressed in a more physically meaningful manner by noting that it is related to the mean curvature, K, of the interface (Hobson et al., 2011).

$$2K = -\nabla \cdot \mathbf{n} \tag{28}$$

2.3 Problem Statement

The system is formulated as in figure 3. The physical parameters motivate the choice of scaling variables. The characteristic lengthscale is chosen to be the sphere radius a, characteristic viscosity that of the upper fluid η_1 and characteristic velocity to be the

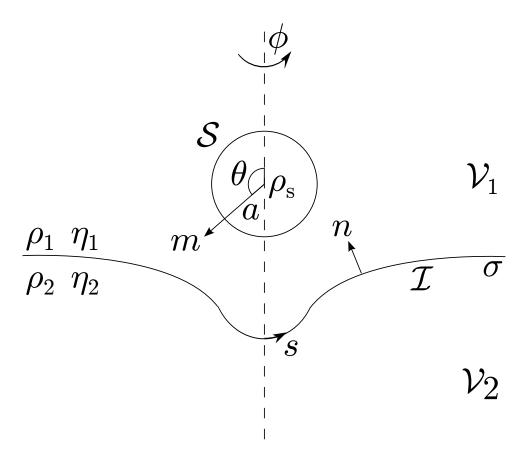


Figure 3: Diagrammatic representation of the system. A sphere falls under gravity, at low Reynolds number, towards an initially horizontal interface between two density stratified, immiscible semi-infinite fluids. See table 1 for definition of symbols.

Stokes velocity (Reynolds, 1886),

$$U_{\rm c} = \frac{2(\rho_{\rm s} - \rho_1)ga^2}{9\eta_1}. (29)$$

This means the capillary and Bond numbers can be expressed as:

$$Ca = \frac{(\rho_s - \rho_1)ga^2}{\sigma},\tag{30}$$

$$Bo = \frac{(\rho_2 - \rho_1)ga^2}{\sigma}.$$
 (31)

The dimensionless stress tensor for each fluid can be written as

$$T'_{\alpha,ij}(\boldsymbol{x'}) = -P'_{d,\alpha}(\boldsymbol{x'})\delta_{ij} + \Lambda_{\alpha}[\partial'_{i}u'_{\alpha,j}(\boldsymbol{x'}) - \partial'_{j}u'_{\alpha,i}(\boldsymbol{x'})]. \tag{32}$$

The parameter Λ_{α} is defined as

$$\Lambda_{\alpha} = \frac{\eta_{\alpha}}{\eta_{1}} = \begin{cases}
1, & \alpha = 1 \\
\frac{\eta_{2}}{\eta_{1}} = \lambda, & \alpha = 2
\end{cases}$$
(33)

Note λ is the viscosity ratio of the two fluids. It is straightforward to apply the general equations of motion and boundary conditions to the problem. The equations of motion, expressed in the Einstein summation convention (Riley et al., 2006) which will be used from now on, appear as

$$\partial_i' u_{\alpha,i}'(\boldsymbol{x'}) = 0, \tag{34}$$

and

$$\partial_{i}^{\prime} T_{\alpha,ij}^{\prime}(\boldsymbol{x}^{\prime}) = 0. \tag{35}$$

Here, $\alpha = 1, 2$ and denotes the fluid, i, j denote components of tensoral quantities.

The first boundary condition that we impose is that the undisturbed fluid is quiescent;

$$u'_{\alpha,i}(\boldsymbol{x'}) \to 0 \text{ as } |\boldsymbol{x'}| \to \infty.$$
 (36)

The kinematic boundary condition on the fluid interface (equation 20) can be expressed as

$$u'_{1,i}(\boldsymbol{x'}) = u'_{2,i}(\boldsymbol{x'}), \quad \boldsymbol{x'} \in S_{\text{int}}.$$
 (37)

The dynamic boundary condition is also imposed at the interface;

$$\operatorname{Ca} n_{i}[T'_{1,ij}(\boldsymbol{x'}) - T'_{2,ij}(\boldsymbol{x'})] + \operatorname{Bo}\hat{z}_{i}x'_{i}n_{j} = n_{j}\partial'_{i}n_{i}.$$
(38)

The kinematic boundary condition on the sphere surface is one of no-slip meaning the fluid velocity at the surface has to equal the sphere velocity;

$$u'_{1,i}(\boldsymbol{x'}) = u'_{s,i}, \quad \boldsymbol{x'} \in \mathcal{S}.$$
 (39)

The final boundary condition is the dynamic boundary condition on the sphere. The force on the fluid due to the sphere originates from the balance between gravity and buoyancy;

$$F_i = \frac{4\pi a^3 (\rho_s - \rho_1) g\hat{z}_i}{3}. (40)$$

Substituting this into equation 19 and using equation 29 we obtain

$$\int_{\mathcal{S}} n_i T'_{1,ij}(\boldsymbol{x'}) d\mathcal{S}' = 6\pi \hat{z}_j$$
(41)

The dimensionless numbers that describe the system are the set $\{\lambda, Ca, Bo\}$. However, an equivalent set can be generated by defining the dimensionless density ratio D;

$$D = \frac{\mathrm{Ca}}{\mathrm{Bo}} = \frac{\rho_{\mathrm{s}} - \rho_{\mathrm{1}}}{\rho_{\mathrm{2}} - \rho_{\mathrm{1}}}.\tag{42}$$

Therefore, we can also describe the system with the set $\{\lambda, D, Bo\}$. This allows us to re-express equation 38 as

DBo
$$n_i[T'_{1,ij}(\boldsymbol{x'}) - T'_{2,ij}(\boldsymbol{x'})] = n_j(\partial'_i n_i - \operatorname{Bo}\hat{z}_i x'_i)$$
 (43)

To summarise, the problem is completely described by equations 34 to 37, and equations 39, 41 and 43.

2.4 Greens functions

In order to derive the boundary integral equations, it is necessary to make use of the Greens functions (Riley et al., 2006) for Stokes flow, $\hat{u}_{\alpha,i}(\mathbf{x'} - \mathbf{y'})$ and $\hat{T}_{\alpha,ij}(\mathbf{x'} - \mathbf{y'})$, defined such that

$$\partial_i' \hat{u}_{\alpha,i}(\mathbf{x'} - \mathbf{y'}) = 0 \tag{44}$$

and

$$\partial_i' \hat{T}_{\alpha,ij}(\mathbf{x'} - \mathbf{y'}) + \mathcal{F}_j \delta(\mathbf{x'} - \mathbf{y'}) = 0, \tag{45}$$

where \mathcal{F} is a arbitrary constant vector. Equations 44 and 45 can be solved following Ladyzhenskaya (1963) using Fourier transforms (appendix A) to show that

$$\hat{u}_{\alpha,j}(\boldsymbol{\xi}) = \frac{\mathcal{F}_i J_{ij}(\boldsymbol{\xi})}{\Lambda_{\alpha}},\tag{46}$$

and

$$\hat{T}_{\alpha,ij}(\boldsymbol{\xi}) = K_{ijk}(\boldsymbol{\xi})\mathcal{F}_k,\tag{47}$$

where $\boldsymbol{\xi} = \boldsymbol{x'} - \boldsymbol{y'}$ and

$$J_{ij}(\boldsymbol{\xi}) = \frac{1}{8\pi |\boldsymbol{\xi}|} \left(\delta_{ij} + \frac{\xi_i \xi_j}{|\boldsymbol{\xi}|^2} \right), \tag{48}$$

and

$$K_{ijk}(\boldsymbol{\xi}) = \frac{-3\xi_i \xi_j \xi_k}{4\pi |\boldsymbol{\xi}|^5} \tag{49}$$

2.5 Integral Representation of Stokes Equations

We now substitute these expressions for the Greens functions into the Lorentz Reciprocal Theorem (equation 116 in appendix D) and simplify using equations 35 and 45 to find

$$\int_{\mathcal{V}_{\alpha}} u'_{\alpha,k}(\boldsymbol{x'}) \delta(\boldsymbol{\xi}) d\boldsymbol{x'}^{3} = \frac{1}{\Lambda_{\alpha}} \int_{\mathcal{S}_{\alpha}} J_{ik}(\boldsymbol{\xi}) T'_{\alpha,ij}(\boldsymbol{x'}) n_{\alpha,j}(\boldsymbol{x'}) d\boldsymbol{x'}^{2} - \int_{\mathcal{S}_{\alpha}} u'_{\alpha,i}(\boldsymbol{x'}) K_{ijk}(\boldsymbol{\xi}) n_{\alpha,j}(\boldsymbol{x'}) d\boldsymbol{x'}^{2}. \tag{50}$$

Finally make the transformation $x' \leftrightarrow y'$ and use the symmetry properties of the kernels (equations 86 and 87 in appendix A) and the delta function (equation 106 in appendix B) to obtain the general form of the integral representation of the Stokes equations;

$$\int_{\mathcal{V}_{\alpha}} u'_{\alpha,k}(\mathbf{y'}) \delta(\mathbf{\xi}) d\mathbf{y'}^{3} = \frac{1}{\Lambda_{\alpha}} \int_{\mathcal{S}_{\alpha}} J_{ik}(\mathbf{\xi}) T'_{\alpha,ij}(\mathbf{y'}) n_{\alpha,j}(\mathbf{y'}) d\mathbf{y'}^{2} + \int_{\mathcal{S}_{\alpha}} u'_{\alpha,i}(\mathbf{y'}) K_{ijk}(\mathbf{\xi}) n_{\alpha,j}(\mathbf{y'}) d\mathbf{y'}^{2}.$$
(51)

Using the definition of the delta function (equation 104 in appendix B) this means

$$\frac{1}{\Lambda_{\alpha}} \int_{\mathcal{S}_{\alpha}} J_{ik}(\boldsymbol{\xi}) T'_{\alpha,ij}(\boldsymbol{y'}) n_{\alpha,j}(\boldsymbol{y'}) d\boldsymbol{y'}^{2} + \int_{\mathcal{S}_{\alpha}} u'_{\alpha,i}(\boldsymbol{y'}) K_{ijk}(\boldsymbol{\xi}) n_{\alpha,j}(\boldsymbol{y'}) d\boldsymbol{y'}^{2} = \begin{cases} u'_{\alpha,k}(\boldsymbol{x'}) & \boldsymbol{x'} \in \mathcal{V}_{\alpha} \\ \frac{u'_{\alpha,k}(\boldsymbol{x'})}{2} & \boldsymbol{x'} \in \mathcal{S}_{\alpha} \\ 0 & \text{otherwise} \end{cases}$$
(52)

This equation needs to be considered for each fluid separately. For fluid 1 ($\alpha = 1$), $S_1 = S + I$, $n_1(y') = m(y')$ for $y' \in S$ and $n_1(y') = n(y')$ for $y' \in I$. Noting that for

 $\mathbf{y'} \in \mathcal{S}, \ u'_{1,i}(\mathbf{y'}) = u_{s,i} \text{ and that } \partial'_j K_{ijk}(\boldsymbol{\xi}) = -\delta_{ik}\delta(\boldsymbol{\xi}) \text{ (as must follow from equations 45}$ and 89) the boundary integral equation for fluid 1 can be written as

$$\int_{\mathcal{S}} J_{ik}(\boldsymbol{\xi}) T'_{1,ij}(\boldsymbol{y'}) m_j(\boldsymbol{y'}) d^2 \boldsymbol{y'} + \int_{\mathcal{I}} J_{ik}(\boldsymbol{\xi}) T'_{1,ij}(\boldsymbol{y'}) n_j(\boldsymbol{y'}) d^2 \boldsymbol{y'} + \int_{\mathcal{I}} u'_{1,i}(\boldsymbol{y'}) K_{ijk}(\boldsymbol{\xi}) n_j(\boldsymbol{y}) d^2 \boldsymbol{y'} = \begin{cases} \frac{u'_{1,k}(\boldsymbol{x'})}{2} & \boldsymbol{x'} \in \mathcal{I} \\ u'_{s,k} & \boldsymbol{x'} \in \mathcal{S} \end{cases}$$
(53)

For fluid 2, $S_2 = \mathcal{I}$ and $n_2(y') = -n(x')$ for $x' \in \mathcal{I}$. Using equation 37 the boundary integral equation for fluid 2 can be written as

$$-\int_{\mathcal{I}} J_{ik}(\boldsymbol{\xi}) T'_{1,ij}(\boldsymbol{y'}) n_j(\boldsymbol{y'}) d^2 \boldsymbol{y'} - \lambda \int_{\mathcal{I}} u'_{1,i}(\boldsymbol{y'}) K_{ijk}(\boldsymbol{\xi}) n_j(\boldsymbol{y}) d^2 \boldsymbol{y'} = \frac{\lambda u'_{1,k}(\boldsymbol{x'})}{2} \quad \boldsymbol{x'} \in \mathcal{I}. \quad (54)$$

Equations 53 and 54 can be added together and combined with equation 43 to obtain

$$\int_{\mathcal{S}} J_{ik}(\boldsymbol{\xi}) f_{s,i}(\boldsymbol{y'}) d^{2}\boldsymbol{y'} + \frac{9}{2DBo} \int_{\mathcal{I}} J_{ik}(\boldsymbol{\xi}) n_{i}(\boldsymbol{y'}) [\partial'_{j} n_{j}(\boldsymbol{y'}) - \hat{z}_{j} y'_{j} Bo] d^{2}\boldsymbol{y'} + (1 - \lambda) \int_{\mathcal{I}} u'_{1,i}(\boldsymbol{y'}) K_{ijk}(\boldsymbol{\xi}) n_{j}(\boldsymbol{y'}) d^{2}\boldsymbol{y'} = \begin{cases} \frac{(1 + \lambda) u'_{1,k}(\boldsymbol{x'})}{2} & \boldsymbol{x'} \in \mathcal{I} \\ u'_{s,k} & \boldsymbol{x'} \in \mathcal{S} \end{cases} .$$
(55)

This together with equation 41 completely describes the system in an integral representation. Since the system is axisymmetric, it is useful to extract the azimuthal integration from the surface integrals. Following some manipulation (DETAILED IN APPENDIX ??) the integral equations can be expressed as

$$\int_{\theta=0}^{\pi} B_{\alpha\beta}(\boldsymbol{x'}, \theta) b_{\beta}(\theta) \sin \theta d\theta + \int_{s=0}^{\infty} \left(A_{\alpha\beta}(\boldsymbol{x'}, s) y_{r}(s) - \frac{(1+\lambda)\delta_{\alpha\beta}\delta(s-s_{0})}{2} \right) a_{\beta}(s) ds$$

$$= -\int_{s=0}^{\infty} C_{\alpha}(\boldsymbol{x'}, s) y_{r}(s) ds, \quad \text{when } \boldsymbol{x'} \in \mathcal{I},$$
(56)

$$\int_{\theta=0}^{\pi} B_{\alpha\beta}(\boldsymbol{x'}, \theta) b_{\beta}(\theta) \sin \theta d\theta + \int_{s=0}^{\infty} A_{\alpha\beta}(\boldsymbol{x'}, s) a_{\beta}(s) y_{r}(s) ds - d_{\alpha} = \int_{s=0}^{\infty} C_{\alpha}(\boldsymbol{x'}, s) y_{r}(s) ds, \quad \text{when } \boldsymbol{x'} \in \mathcal{S},$$
(57)

and

$$\int_{\theta=0}^{\pi} b_2(\theta) \sin \theta d\theta = 3, \tag{58}$$

where the quantities A, B, C, a, b and d are defined as:

$$\mathbf{A} = (1 - \lambda) \int_{\phi=0}^{2\pi} \begin{pmatrix} n_r(K_{111}\cos^2\phi + K_{221}\sin^2\phi + 2K_{121}\sin\phi\cos\phi) & n_r(K_{131}\cos\phi + K_{231}\sin\phi) \\ + n_z(K_{131}\cos\phi + K_{231}\sin\phi) & + n_zK_{331} \end{pmatrix} d\phi,$$

$$n_r(K_{113}\cos^2\phi + K_{223}\sin^2\phi + 2K_{123}\sin\phi\cos\phi) & n_r(K_{133}\cos\phi + K_{233}\sin\phi) \\ + n_z(K_{133}\cos\phi + K_{233}\sin\phi) & + n_zK_{333} \end{pmatrix} (59)$$

$$\mathbf{B} = \int_{\phi=0}^{2\pi} \begin{pmatrix} J_{11}\cos\phi + J_{21}\sin\phi & J_{31} \\ J_{13}\cos\phi + J_{23}\sin\phi & J_{33} \end{pmatrix} d\phi, \tag{60}$$

$$C = \frac{9(\partial'_{j}n_{j} - \text{Bo}y_{z})}{2D\text{Bo}} \int_{\phi=0}^{2\pi} \begin{pmatrix} n_{r}(J_{11}\cos\phi + J_{21}\sin\phi) + n_{z}J_{31} \\ n_{r}(J_{13}\cos\phi + J_{23}\sin\phi) + n_{z}J_{23} \end{pmatrix} d\phi,$$
(61)

$$\boldsymbol{a} = \begin{pmatrix} u'_{1,r}(s) \\ u'_{1,z}(s) \end{pmatrix},\tag{62}$$

$$\boldsymbol{b} = \begin{pmatrix} f_{s,r}(s) \\ f_{s,z}(s) \end{pmatrix}, \tag{63}$$

and

$$\boldsymbol{d} = \begin{pmatrix} 0 \\ u_{\rm s}' \end{pmatrix} \tag{64}$$

For brevity the function arguments have been dropped from the kernels and the normal vectors but $n_i = n_i(s)$ and in equation 59, $K_{ijk} = K_{ijk}(\mathbf{x'}, s)$, in equation 60, $J_{ij} = J_{ij}(\mathbf{x'}, \theta)$ and in equation 61, $J_{ij} = J_{ij}(\mathbf{x'}, s)$.

A Greens Functions for Stokes Flow

We present here a derivation of equations 46 and 89 following Ladyzhenskaya (1963). First, the Greens function for dynamic pressure $\hat{P}_{\alpha}(\boldsymbol{\xi})$ is defined such that

$$\hat{T}_{\alpha,ij}(\boldsymbol{\xi}) = -\hat{P}_{\alpha}(\boldsymbol{\xi}) + \Lambda_{\alpha}[\partial_i'\hat{u}_{\alpha,j}(\boldsymbol{\xi}) + \partial_i'\hat{u}_{\alpha,i}(\boldsymbol{\xi})]. \tag{65}$$

Substituting this into equation 45 and using equation 44 yields

$$-\partial_{j}'\hat{P}_{\alpha}(\boldsymbol{\xi}) + \Lambda_{\alpha}\partial_{i}'\partial_{i}'\hat{u}_{\alpha,j}(\boldsymbol{\xi}) + \mathcal{F}_{j}\delta(\boldsymbol{\xi}) = 0.$$
 (66)

We also define two further quantities $\bar{P}_{\alpha,i}$ and $\bar{u}_{\alpha,ij}$ such that

$$\hat{P}_{\alpha}(\boldsymbol{\xi}) = \mathcal{F}_i \bar{P}_{\alpha,i}(\boldsymbol{\xi}), \tag{67}$$

and

$$\hat{u}_{\alpha,j}(\boldsymbol{\xi}) = \mathcal{F}_i \bar{u}_{\alpha,ij}(\boldsymbol{\xi}). \tag{68}$$

Substitution of these expressions into equations 66 and 44 and rearranging results in

$$-\partial_{i}'\bar{P}_{\alpha,i}(\boldsymbol{\xi}) + \Lambda_{\alpha}\partial_{k}'\partial_{k}'\bar{u}_{\alpha,ij}(\boldsymbol{\xi}) + \delta_{ij}\delta(\boldsymbol{\xi}) = 0, \tag{69}$$

and

$$\partial_i' \bar{u}_{\alpha,ij}(\boldsymbol{\xi}) = 0. \tag{70}$$

To derive functional forms for the Greens functions it is necessary to express equations 69 and 70 in Fourier representation. To do this we need to define the Fourier transformed variables $\tilde{P}_{\alpha,i}$ and $\tilde{u}_{\alpha,ij}$ Riley et al. (2006):

$$\bar{P}_{\alpha,i}(\boldsymbol{\xi}) = \frac{1}{(2\pi)^{3/2}} \int \tilde{P}_{\alpha,i}(\boldsymbol{k}) e^{i\boldsymbol{k}\cdot\boldsymbol{\xi}} d^3\boldsymbol{k}, \tag{71}$$

and

$$\bar{u}_{\alpha,ij}(\boldsymbol{\xi}) = \frac{1}{(2\pi)^{3/2}} \int \tilde{u}_{\alpha,ij}(\boldsymbol{k}) e^{i\boldsymbol{k}\cdot\boldsymbol{\xi}} d^3\boldsymbol{k}.$$
 (72)

Using these definitions, and equation 107 in appendix B, in equations 69 and 70 gives the Fourier representations of the Stokes equations and the continuity equation. Following some manipulation these can be written as

$$-ik_j \tilde{P}_{\alpha,i}(\mathbf{k}) - \Lambda_{\alpha} k^2 \tilde{u}_{\alpha,ij}(\mathbf{k}) + \frac{\delta_{ij}}{(2\pi)^{3/2}} = 0, \tag{73}$$

and

$$k_i \tilde{u}_{\alpha,ij}(\mathbf{k}) = 0, \tag{74}$$

where $k = |\mathbf{k}|$. By multiplying equation 73 by k_j , substituting in equation 74 and rearranging, it is then possible to obtain the Fourier representation of the Greens function for pressure;

$$\tilde{P}_{\alpha,i}(\mathbf{k}) = \frac{-ik_i}{(2\pi)^{3/2}k^2}.$$
(75)

A final substitution of this into equation 71 gives the Greens function for pressure;

$$\bar{P}_{\alpha,i}(\boldsymbol{\xi}) = \frac{-i}{(2\pi)^3} \int \frac{k_i e^{i\boldsymbol{k}\cdot\boldsymbol{\xi}} d^3\boldsymbol{k}}{k^2}.$$
 (76)

This integral is evaluated in appendix A.1 and it is shown that

$$\bar{P}_{\alpha,i}(\boldsymbol{\xi}) = -\frac{1}{4\pi} \partial_i' \left(\frac{1}{\xi}\right) = \frac{\xi_i}{4\pi \xi^3} \quad , \quad \xi = \xi_i \xi_i. \tag{77}$$

We also need to find an equivalent expression for $\bar{u}_{\alpha,ij}$. To do so, substitute equation 75 into equation 73 and rearrange;

$$\tilde{u}_{\alpha,ij}(\mathbf{k}) = \frac{k^2 \delta_{ij} - k_i k_j}{(2\pi)^{3/2} k^4 \Lambda_{\alpha}}.$$
(78)

Combining this with equation 72 results in an expression for the Greens function for velocity;

$$\bar{u}_{\alpha,ij}(\boldsymbol{\xi}) = \frac{1}{(2\pi)^3 \Lambda_{\alpha}} \left(\delta_{ij} \int \frac{e^{i\boldsymbol{k}\cdot\boldsymbol{\xi}} d^3\boldsymbol{k}}{k^2} - \int \frac{k_i k_j e^{i\boldsymbol{k}\cdot\boldsymbol{\xi}} d^3\boldsymbol{k}}{k^4} \right).$$
 (79)

These integrals are evaluated in appendix A.2 (equations 97 and 103) and following some manipulation we find

$$\bar{u}_{\alpha,ij}(\boldsymbol{\xi}) = \frac{1}{8\pi\Lambda_{\alpha}\xi} \left(\delta_{ij} + \frac{\xi_i \xi_j}{\xi^2} \right). \tag{80}$$

We can now substitute equations 77 and 80 into 67 and 68 to obtain

$$\hat{P}_{\alpha}(\boldsymbol{\xi}) = \frac{\mathcal{F}_{i}\xi_{i}}{4\pi\xi^{3}},\tag{81}$$

and

$$\hat{u}_{\alpha,j}(\boldsymbol{\xi}) = \frac{\mathcal{F}_i}{8\pi\Lambda_{\alpha}\xi} \left(\delta_{ij} + \frac{\xi_i\xi_j}{\xi^2}\right). \tag{82}$$

Substitution of equations 81 and 82 into equation 65 results in

$$\hat{T}_{\alpha,ij}(\boldsymbol{\xi}) = \frac{-3\mathcal{F}_k \xi_i \xi_j \xi_k}{4\pi \xi^5} \tag{83}$$

The kernels J and K are defined as

$$J_{ij} = \frac{1}{8\pi\xi} \left(\delta_{ij} + \frac{\xi_i \xi_j}{\xi^2} \right) \tag{84}$$

$$K_{ijk} = \frac{-3\xi_i \xi_j \xi_k}{4\pi \xi^5} \tag{85}$$

Note that under the interchange $\xi \to -\xi$ the kernels are symmetric and antisymmetric respectively

$$J_{ki}(-\boldsymbol{\xi}) = J_{ki}(\boldsymbol{\xi}) \tag{86}$$

$$K_{jik}(-\boldsymbol{\xi}) = -K_{jik}(\boldsymbol{\xi}) \tag{87}$$

This means we can express the velocity and stress Greens functions as

$$\hat{u}_{\alpha,j}(\boldsymbol{\xi}) = \frac{\mathcal{F}_i J_{ij}(\boldsymbol{\xi})}{\Lambda_{\alpha}} \tag{88}$$

$$\hat{T}_{\alpha,ij}(\boldsymbol{\xi}) = K_{ijk}(\boldsymbol{\xi})\mathcal{F}_k \tag{89}$$

A.1 Integral for Greens Function for Pressure

To evaluate the integral in equation 76 first define a volume \mathcal{V} bounded by a surface \mathcal{S} . The consider

$$\int_{\mathcal{V}} \nabla^2 \left(\frac{1}{\xi} \right) d^3 \boldsymbol{\xi} = \int_{\mathcal{V}} \boldsymbol{\nabla} \cdot \boldsymbol{\nabla} \left(\frac{1}{\xi} \right) d^3 \boldsymbol{\xi}. \tag{90}$$

Using the divergence theorem (appendix C) this can be written as

$$\int_{\mathcal{V}} \nabla^2 \left(\frac{1}{\xi} \right) d^3 \boldsymbol{r} = \int_{\mathcal{S}} \boldsymbol{\nabla} \left(\frac{1}{\xi} \right) \cdot d\mathcal{S}. \tag{91}$$

This can be evaluated for the case that \mathcal{V} is a sphere of radius R;

$$\int_{\mathcal{V}} \nabla^2 \left(\frac{1}{\xi} \right) d^3 \boldsymbol{\xi} = -4\pi = -4\pi \int_{\mathcal{V}} \delta(\boldsymbol{\xi}) d^3 \boldsymbol{\xi}, \tag{92}$$

where the final step follows from comparison with equation 105. Using equation 107 leads to

$$\nabla^2 \left(\frac{1}{\xi} \right) = \frac{-4\pi}{(2\pi)^3} \int e^{i\mathbf{k}\cdot\boldsymbol{\xi}} d^3\mathbf{k}. \tag{93}$$

Inspection of this then suggests

$$\partial_i \left(\frac{1}{\xi} \right) = \frac{4i\pi}{(2\pi)^3} \int \frac{k_i e^{i\mathbf{k}\cdot\boldsymbol{\xi}} d^3\mathbf{k}}{k^2}.$$
 (94)

Hence

$$\frac{-i}{(2\pi)^3} \int \frac{k_i e^{i\mathbf{k}\cdot\boldsymbol{\xi}} d^3\mathbf{k}}{k^2} = -\frac{1}{4\pi} \partial_i \left(\frac{1}{\xi}\right). \tag{95}$$

A.2 Integrals for the Greens Function for Velocity

Equation 79 contains two integrals that need to be evaluated. By inspecting equation 93 it can be seen

$$\frac{1}{\xi} = \frac{4\pi}{(2\pi)^3} \int \frac{e^{i\mathbf{k}\cdot\boldsymbol{\xi}} d^3\mathbf{k}}{k^2}.$$
 (96)

Hence

$$\int \frac{e^{i\boldsymbol{k}\cdot\boldsymbol{\xi}}d^3\boldsymbol{k}}{k^2} = \frac{(2\pi)^3}{4\pi\xi}.$$
 (97)

The second integral requires a bit more work. Firstly, express it in a different form;

$$\int \frac{k_i k_j e^{i\mathbf{k}\cdot\boldsymbol{\xi}} d^3 \mathbf{k}}{k^4} = \partial_i \partial_j \left(\int \frac{e^{i\mathbf{k}\cdot\boldsymbol{\xi}} d^3 \mathbf{k}}{k^4} \right). \tag{98}$$

To evaluate this, first consider $\nabla^4 \xi = \nabla^2 (\nabla^2 \xi)$. Expanding ∇^2 in spherical polar coordinates centred on $\xi = 0$ shows

$$\nabla^4 \xi = 2\nabla^2 \left(\frac{1}{\xi}\right). \tag{99}$$

Combining this with equation 93 we obtain

$$\nabla^4 \xi = \frac{-8\pi}{(2\pi)^3} \int e^{i\boldsymbol{k}\cdot\boldsymbol{\xi}} d^3 \boldsymbol{k}.$$
 (100)

Inserection of this yields

$$\xi = \frac{-8\pi}{(2\pi)^3} \int \frac{e^{i\mathbf{k}\cdot\boldsymbol{\xi}} d^3\mathbf{k}}{k^4}.$$
 (101)

Rearrangement of this produces an expression for the integral on the right hand side of equation 98;

$$\int \frac{e^{i\mathbf{k}\cdot\boldsymbol{\xi}}d^3\mathbf{k}}{k^4} = -\frac{(2\pi)^3\xi}{8\pi}.$$
 (102)

Hence

$$\int \frac{k_i k_j e^{i\mathbf{k}\cdot\boldsymbol{\xi}} d^3 \mathbf{k}}{k^4} = \frac{(2\pi)^3 \partial_i' \partial_j' \xi}{8\pi}.$$
 (103)

B Dirac Delta Function

In a volume V bounded by a surface S, the Dirac delta function $\delta(x - y)$ is defined as (Riley et al., 2006)

$$\int_{\mathcal{V}} f(\boldsymbol{y}) \delta(\boldsymbol{x} - \boldsymbol{y}) d^{3} \boldsymbol{y} = \begin{cases}
f(\boldsymbol{x}) & \boldsymbol{x} \in \mathcal{V} \\
\frac{f(\boldsymbol{x})}{2} & \boldsymbol{x} \in \mathcal{S} \\
0 & \text{otherwise}
\end{cases}$$
(104)

The result for $x \in \mathcal{S}$ is only valied for the case that the surface is Lyapunov smooth (a local tangent plane exists everywhere) (REFERENCE FOR LYAPUNOV SMOOTH

SURFACE). Equation 104 means that

$$\int_{\mathcal{V}} \delta(\boldsymbol{x} - \boldsymbol{y}) d^3 \boldsymbol{y} = 1 \quad \boldsymbol{x} \in \mathcal{V}.$$
 (105)

A key property of the delta function is that it is symmetric under a change of sign of the argument

$$\delta(-\boldsymbol{x}) = \delta(\boldsymbol{x}) \tag{106}$$

It also needs to be noted that the Dirac delta function can be expressed as (Riley et al., 2006)

$$\delta(\boldsymbol{\xi}) = \frac{1}{(2\pi)^3} \int e^{i\boldsymbol{k}\cdot\boldsymbol{\xi}} d^3\boldsymbol{k}$$
 (107)

C Divergence Theorem

The divergence theorem states that for a volume \mathcal{V} bounded by a surface \mathcal{S} , then for a continuous and differentiable vector field \boldsymbol{a} Riley et al. (2006)

$$\int_{\mathcal{V}} \nabla \cdot \boldsymbol{a} d\mathcal{V} = \oint_{\mathcal{S}} \boldsymbol{a} \cdot d\boldsymbol{S}$$
 (108)

D Lorentz Reciprocal Theorem

Here we present a proof of the Lorentz reciprocal theorem following Kim and Karrila (2005). The dimensionless strain field in fluid alpha is defined as

$$e'_{\alpha,ij}(\boldsymbol{\xi}) = \frac{\partial'_i u'_{\alpha,j}(\boldsymbol{\xi}) + \partial'_j u'_{\alpha,i}(\boldsymbol{\xi})}{2},\tag{109}$$

and the corresponding Greens function as

$$\hat{e}_{\alpha,ij}(\boldsymbol{\xi}) = \frac{\partial_i' \hat{u}_{\alpha,j}(\boldsymbol{\xi}) + \partial_j' \hat{u}_{\alpha,i}(\boldsymbol{\xi})}{2}.$$
(110)

This allows the Greens function for the stress tensor to be expressed as

$$\hat{T}_{\alpha,ij}(\boldsymbol{\xi}) = \hat{P}_{\alpha}(\boldsymbol{\xi}) + 2\hat{e}_{\alpha,ij}(\boldsymbol{\xi}). \tag{111}$$

Now consider the double contraction of the dimensionless stress tensor and strain Green function tensor;

$$T'_{\alpha,ij}(\boldsymbol{x})\hat{e}_{\alpha,ij}(\boldsymbol{\xi}) = -P'_{d,\alpha}(\boldsymbol{x})\hat{e}_{\alpha,ii}(\boldsymbol{\xi}) + 2e'_{\alpha,ij}(\boldsymbol{x})\hat{e}_{\alpha,ij}(\boldsymbol{\xi}) = 2e'_{\alpha,ij}(\boldsymbol{x})\hat{e}_{\alpha,ij}(\boldsymbol{\xi}).$$
(112)

The last step here follows by using equations 110 and 44 to show that the first term on the right hand side is equal to zero. In an analogous fashion it can be shown that

$$\hat{T}_{\alpha,ij}(\boldsymbol{\xi})e'_{\alpha,ij}(\boldsymbol{x}) = 2\hat{e}_{\alpha,ij}(\boldsymbol{\xi})e'_{\alpha,ij}(\boldsymbol{x}). \tag{113}$$

Since the right hand sides of equations 112 and 113 are identical this means

$$T'_{\alpha,ij}(\boldsymbol{x})\hat{e}_{\alpha,ij}(\boldsymbol{\xi}) = \hat{T}_{\alpha,ij}(\boldsymbol{\xi})e'_{\alpha,ij}(\boldsymbol{x}). \tag{114}$$

Expanding the strain tensors in terms of the velocity gradients and integrating over a volume \mathcal{V} bounded by a surface \mathcal{S} with respect to \boldsymbol{x} leads to

$$\int_{\mathcal{V}} \partial_j' [T'_{ij}(\boldsymbol{x'}) \hat{u}_i(\boldsymbol{\xi})] d\boldsymbol{x'}^3 - \int_{\mathcal{V}} [\partial_j' T'_{ij}(\boldsymbol{x'})] \hat{u}_i(\boldsymbol{\xi}) d\boldsymbol{x'}^3 = \int_{\mathcal{V}} \partial_j' [\hat{T}_{ij}(\boldsymbol{\xi}) u'_i(\boldsymbol{x'})] d\boldsymbol{x'}^3 - \int_{\mathcal{V}} [\partial_j' \hat{T}_{ij}(\boldsymbol{\xi})] u'_i(\boldsymbol{x'}) d\boldsymbol{x'}^3. \tag{115}$$

REFERENCES REFERENCES

Here the integration has been defined in the sense of the Cauchy Principle Value (CPV) (Riley et al., 2006) to allow for the singular point in the Greens functions at $\boldsymbol{\xi} = \mathbf{0}$. Defining \boldsymbol{n} as the normal to the surface \mathcal{S} , the divergence theorem (appendix C) can then to applied to the first term on both sides;

$$\int_{\mathcal{S}} n_j(\boldsymbol{x'}) T'_{ij}(\boldsymbol{x'}) \hat{u}_i(\boldsymbol{\xi}) d\boldsymbol{x'}^2 - \int_{\mathcal{V}} [\partial'_j T'_{ij}(\boldsymbol{x'})] \hat{u}_i(\boldsymbol{\xi}) d\boldsymbol{x'}^3 = \int_{\mathcal{S}} n_j(\boldsymbol{x'}) \hat{T}_{ij}(\boldsymbol{\xi}) u'_i(\boldsymbol{x'}) d\boldsymbol{x'}^2 - \int_{\mathcal{V}} [\partial'_j \hat{T}_{ij}(\boldsymbol{\xi})] u'_i(\boldsymbol{x'}) d\boldsymbol{x'}^3. \tag{116}$$

This is the Lorentz reciprocal theorem with the additional caveat that the integrals have been defined as CPV integrals so that the singular nature of the Greens functions can be dealt with.

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