

# COMPUTATIONAL MATERIAL MODELS

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2013/2014

June 19, 2014

# INDEX

[back to index](#)

- Homogeneous truss
  - Finite element method
  - Nonlinear deformation
  - Weighted residual formulation
  - Finite element method
- 
- One-dimensional material behavior
  - Elastic
  - Elastoplastic
  - Linear viscoelastic
  - Creep
  - Viscoplastic
  - Nonlinear viscoelastic

- Vectors and tensors
  - Kinematics
  - Small (linear) deformation
  - Stress
  - Balance laws
  - Constitutive equations
  - Linear elastic material
  - Material symmetry
  - Linear elastic isotropic material : engineering parameters
  - Linear elastic isotropic material : tensorial form
  - Thermo-elasticity
  - Planar deformation
  - Elastic limit
- 
- Governing equations
  - Solution strategies
  - Weighted residual formulation
- 
- Finite element method
  - Numerical solutions

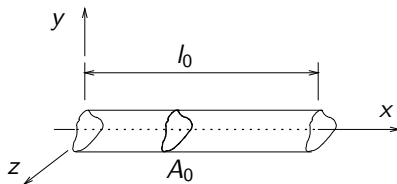
- Three-dimensional material models
  - Elastic
  - Elastoplastic
  - Linear viscoelastic
  - Viscoplastic
  - Nonlinear viscoelastic

- APPENDICES
- Tutorial ; tr2d.m
- Utilities m2cc.m and m2mm.m
- Stiffness and compliance matrices
- WR for axi-symmetric deformation
- FEM for axi-symmerric deformation
- FEM for planar deformation

# HOMOGENEOUS TRUSS

[back to index](#)

# Homogeneous truss



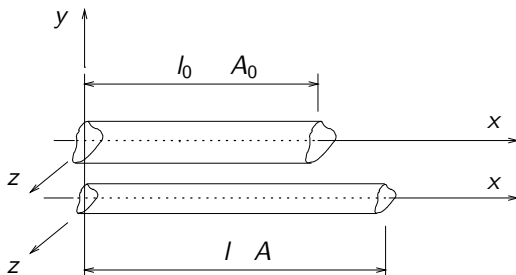
length  
truss cylindrical  $\rightarrow$   
cross-sectional area uniform

$l_0$

$A_0$



# Elongation and contraction



elongation factor

$$\lambda = \frac{l}{l_0} = \frac{l_0 + \Delta l}{l_0} = 1 + \frac{\Delta l}{l_0}$$

contraction

$$\mu = \sqrt{\frac{A}{A_0}}$$

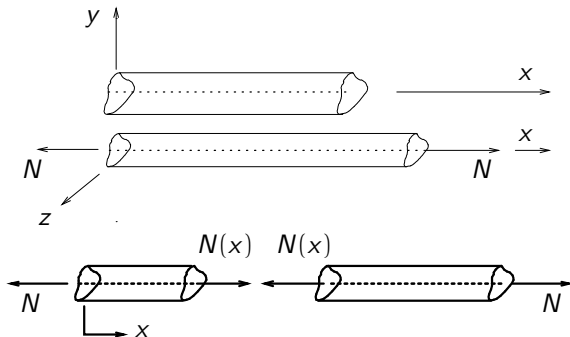
volume change

$$J = \frac{lA}{l_0A_0} = \lambda\mu^2$$

exempl. circular cross section

$$\rightarrow \mu = \frac{d}{d_0} = \sqrt{\frac{A}{A_0}}$$

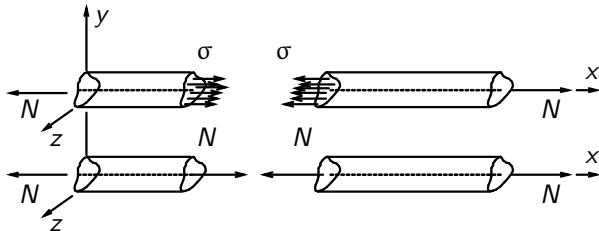
# Stress



axial tensile force (external)  
cross-sectional force (internal)

$$N$$
$$N(x) = N$$

# Axial stress



axial stress

cross-sectional force

stress uniform in cross-section

true stress

engineering stress

relation

$$\sigma = \sigma(y, z)$$

$$N(x) = N = \int_A \sigma(y, z) dA$$

$$N = \int_A \sigma dA = \sigma A$$

$$\sigma = \frac{N}{A}$$

$$\sigma_n = \frac{N}{A_0}$$

$$\sigma = \frac{N}{A} = \frac{A_0}{A} \frac{N}{A_0} = \frac{1}{\mu^2} \sigma_n$$

# Linear elastic behavior

axial stress  $\sim$  strain  $\sigma = E \varepsilon$

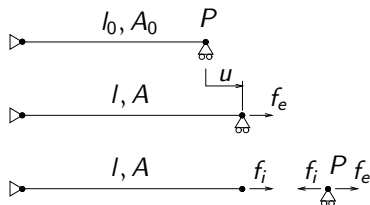
contraction strain  $\varepsilon = \lambda - 1 \rightarrow$

$$\varepsilon_d = \mu - 1 = -\nu \varepsilon = -\nu(\lambda - 1)$$

volume change  $J = (\varepsilon + 1)(-\nu \varepsilon + 1)^2 \approx \varepsilon(1 - 2\nu) + 1$

material	$E$ [GPa]	$\nu$ [-]	material	$E$ [GPa]	$\nu$ [-]
Aluminum	69 - 79	0.31 - 0.34	Copper	105 - 150	0.33 - 0.35
Cast iron	105 - 150	0.21 - 0.30	Steel	200	0.33
Stainless steel	190 - 200	0.28	Lead	14	0.43
Magnesium	41 - 45	0.29 - 0.35	Nickel	180 - 215	0.31
Titanium	80 - 130	0.31 - 0.34	Tungsten	400	0.27
Diamond	820 - 1050	-	Graphite	240 - 390	-
Glass	70 - 80	0.24	Epoxy	3.5 - 17	0.34
Nylon	1.4 - 2.8	0.32 - 0.40	Rubber	0.01 - 0.1	0.5

# Equilibrium



external force

$$f_e$$

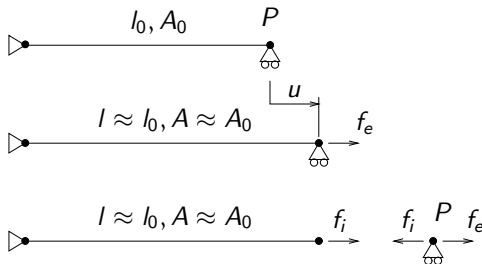
internal force

$$f_i = f_i(u)$$

equilibrium of point  $P$

$$f_i(u) = f_e$$

# Linear deformation



external force

$f_e$

internal force

$f_i = \sigma_n A_0$

equilibrium of point  $P$

$f_i(u) = f_e$

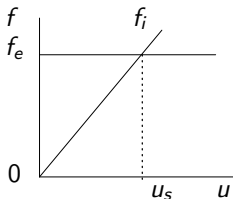
$f_i(u)$  linear

direct solution possible

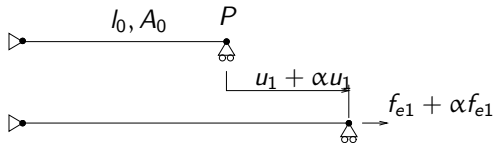
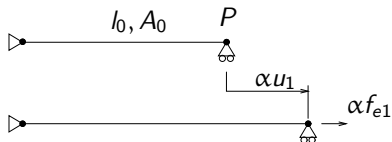
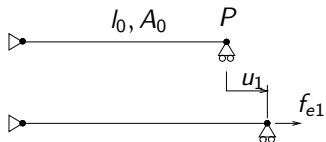
# Direct solution

$$f_i = \sigma_n A_0 = E \varepsilon A_0 = \frac{EA_0}{l_0} u = Ku$$

$$f_i = f_e \rightarrow Ku = f_e \rightarrow u = u_s = \frac{f_e}{K} = \frac{l_0}{EA_0} f_e$$



# Proportionality and superposition

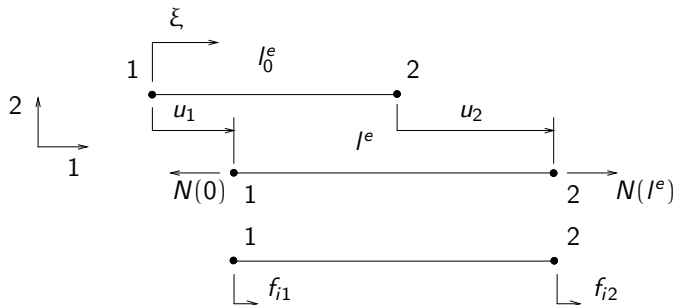




# FINITE ELEMENT METHOD

[back to index](#)

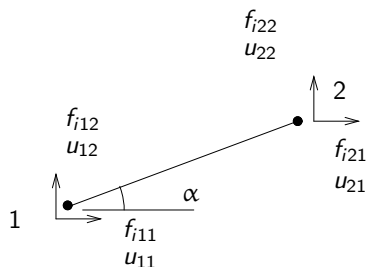
# Truss element



$$\tilde{u}^e = \begin{bmatrix} u_1 \\ u_2 \end{bmatrix} = \begin{bmatrix} u(0) \\ u(l^e) \end{bmatrix}$$

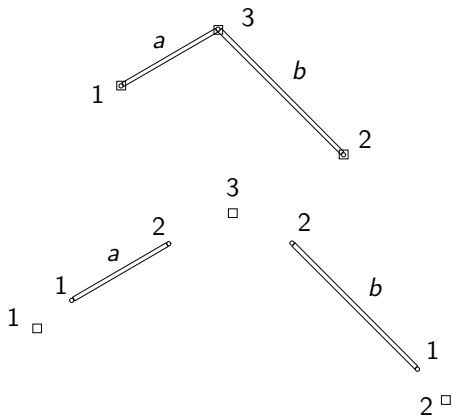
$$\tilde{f}_i^e = \begin{bmatrix} f_{i1} \\ f_{i2} \end{bmatrix} = \begin{bmatrix} -N(0) \\ N(l^e) \end{bmatrix} = \begin{bmatrix} -k(u_2 - u_1) \\ k(u_2 - u_1) \end{bmatrix} = \begin{bmatrix} k & -k \\ -k & k \end{bmatrix} \begin{bmatrix} u_1 \\ u_2 \end{bmatrix}$$

# Two-dimensional element

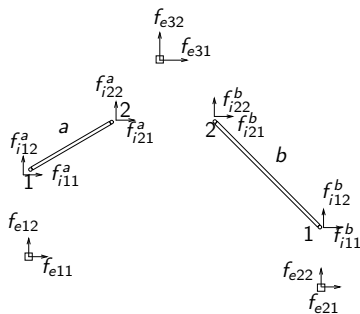


$$\begin{aligned}
 \begin{bmatrix} f_{i11} \\ f_{i12} \\ f_{i21} \\ f_{i22} \end{bmatrix} &= \begin{bmatrix} cf_{i1}^L \\ sf_{i1}^L \\ cf_{i2}^L \\ sf_{i2}^L \end{bmatrix} = k(u_2^L - u_1^L) \begin{bmatrix} -c \\ -s \\ c \\ s \end{bmatrix} = k(u_{21}c + u_{22}s - u_{11}c - u_{12}s) \begin{bmatrix} -c \\ -s \\ c \\ s \end{bmatrix} \\
 &= k \begin{bmatrix} c^2 & cs & -c^2 & -cs \\ cs & s^2 & -cs & -s^2 \\ -c^2 & -cs & c^2 & cs \\ -cs & -s^2 & cs & s^2 \end{bmatrix} \begin{bmatrix} u_{11} \\ u_{12} \\ u_{21} \\ u_{22} \end{bmatrix} = \underline{K}^e \underline{u}^e \quad \begin{cases} c = \cos(\alpha) \\ s = \sin(\alpha) \end{cases}
 \end{aligned}$$

# Assembling



# Assembling : internal forces



$$\begin{bmatrix} f_{e11} \\ f_{e12} \\ f_{e21} \\ f_{e22} \\ f_{e31} \\ f_{e32} \end{bmatrix} = \begin{bmatrix} f_{i11} \\ f_{i12} \\ f_{i21} \\ f_{i22} \\ f_{i31} \\ f_{i32} \end{bmatrix} = \begin{bmatrix} f_{i11}^a \\ f_{i12}^a \\ f_{i11}^b \\ f_{i12}^b \\ f_{i21}^a + f_{i21}^b \\ f_{i22}^a + f_{i22}^b \end{bmatrix} = \begin{bmatrix} f_{i11}^a \\ f_{i12}^a \\ 0 \\ 0 \\ f_{i21}^a \\ f_{i22}^a \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \\ f_{i11}^b \\ f_{i12}^b \\ f_{i21}^b \\ f_{i22}^b \end{bmatrix}$$

## Assembling : nodal displacements

$$\begin{bmatrix} f_{i11} \\ f_{i12} \\ f_{i21} \\ f_{i22} \\ f_{i31} \\ f_{i32} \end{bmatrix} = \begin{bmatrix} k^a c^2 & k^a cs & 0 & 0 & -k^a c^2 & -k^a cs \\ k^a cs & k^a s^2 & 0 & 0 & -k^a cs & -k^a s^2 \\ 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 \\ -k^a c^2 & -k^a cs & 0 & 0 & k^a c^2 & k^a cs \\ -k^a cs & -k^a s^2 & 0 & 0 & k^a cs & k^a s^2 \end{bmatrix} \begin{bmatrix} u_{11} \\ u_{12} \\ u_{21} \\ u_{22} \\ u_{31} \\ u_{32} \end{bmatrix} + \begin{bmatrix} 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & k^b c^2 & k^b cs & -k^b c^2 & -k^b cs \\ 0 & 0 & k^b cs & k^b s^2 & -k^b cs & -k^b s^2 \\ 0 & 0 & -k^b c^2 & -k^b cs & k^b c^2 & k^b cs \\ 0 & 0 & -k^b cs & -k^b s^2 & k^b cs & k^b s^2 \end{bmatrix} \begin{bmatrix} u_{11} \\ u_{12} \\ u_{21} \\ u_{22} \\ u_{31} \\ u_{32} \end{bmatrix} = \begin{bmatrix} k^a c^2 & k^a cs & 0 & 0 & -k^a c^2 & -k^a cs \\ k^a cs & k^a s^2 & 0 & 0 & -k^a cs & -k^a s^2 \\ 0 & 0 & k^b c^2 & k^b cs & -k^b c^2 & -k^b cs \\ 0 & 0 & k^b cs & k^b s^2 & -k^b cs & -k^b s^2 \\ -k^a c^2 & -k^a cs & -k^b c^2 & -k^b cs & k^a c^2 + k^b c^2 & k^a cs + k^b cs \\ -k^a cs & -k^a s^2 & -k^b cs & -k^b s^2 & k^a cs + k^b cs & k^a s^2 + k^b s^2 \end{bmatrix} \begin{bmatrix} u_{11} \\ u_{12} \\ u_{21} \\ u_{22} \\ u_{31} \\ u_{32} \end{bmatrix}$$

Assembled system equations  $\underline{f}_i = \underline{K} \underline{u}$

# Boundary conditions

equilibrium

$$\underline{f}_i = \underline{f}_e \quad \rightarrow \quad \underline{K} \underline{u} = \underline{f}_e = \underline{f}$$

rigid translation

$$\underline{u} = \underline{a}$$

no forces needed

$$\underline{K} \underline{a} = \underline{0} \quad \text{with} \quad \underline{a} \neq \underline{0} \quad \rightarrow \quad \underline{K} \text{ singular}$$

## Prescribed nodal displacements and forces

reorganizing 
$$\underline{u} = \begin{bmatrix} \underline{u}_u \\ \underline{u}_p \end{bmatrix} \quad ; \quad \underline{f} = \begin{bmatrix} \underline{f}_u \\ \underline{f}_p \end{bmatrix}$$



# Partitioning for boundary conditions

reorganizing  $\underline{u} = \begin{bmatrix} \underline{u}_u \\ \underline{u}_p \end{bmatrix} \quad ; \quad \underline{f} = \begin{bmatrix} \underline{f}_u \\ \underline{f}_p \end{bmatrix}$

equilibrium  $\underline{K}\underline{u} = \underline{f}$

partitioning  $\begin{bmatrix} \underline{K}_{uu} & \underline{K}_{up} \\ \underline{K}_{pu} & \underline{K}_{pp} \end{bmatrix} \begin{bmatrix} \underline{u}_u \\ \underline{u}_p \end{bmatrix} = \begin{bmatrix} \underline{f}_u \\ \underline{f}_p \end{bmatrix} \rightarrow$

$$\left. \begin{aligned} \underline{K}_{uu}\underline{u}_u + \underline{K}_{up}\underline{u}_p &= \underline{f}_u \\ \underline{K}_{pu}\underline{u}_u + \underline{K}_{pp}\underline{u}_p &= \underline{f}_p \end{aligned} \right\}$$

solving  $\underline{u}_u$   $\underline{K}_{uu}\underline{u}_u = \underline{f}_u - \underline{K}_{up}\underline{u}_p \rightarrow \underline{u}_u = \underline{K}_{uu}^{-1}(\underline{f}_u - \underline{K}_{up}\underline{u}_p)$

calculating  $\underline{f}_p$   $\underline{f}_p = \underline{K}_{pu}\underline{u}_u + \underline{K}_{pp}\underline{u}_p$

# Links

equilibrium

$$\underline{K} \underline{u} = \underline{f}$$

partitioning

$$\begin{bmatrix} \underline{K}_{ff} & \underline{K}_{fr} & \underline{K}_{fl} \\ \underline{K}_{rf} & \underline{K}_{rr} & \underline{K}_{rl} \\ \underline{K}_{lf} & \underline{K}_{lr} & \underline{K}_{ll} \end{bmatrix} \begin{bmatrix} \underline{u}_f \\ \underline{u}_r \\ \underline{u}_l \end{bmatrix} = \begin{bmatrix} \underline{f}_f \\ \underline{f}_r + \bar{\underline{f}}_r \\ \underline{f}_l + \bar{\underline{f}}_l \end{bmatrix}$$

$$\underline{K}_{ff} \underline{u}_f + \underline{K}_{fr} \underline{u}_r + \underline{K}_{fl} \underline{u}_l = \underline{f}_f$$

$$\underline{K}_{rf} \underline{u}_f + \underline{K}_{rr} \underline{u}_r + \underline{K}_{rl} \underline{u}_l = \underline{f}_r + \bar{\underline{f}}_r$$

$$\underline{K}_{lf} \underline{u}_f + \underline{K}_{lr} \underline{u}_r + \underline{K}_{ll} \underline{u}_l = \underline{f}_l + \bar{\underline{f}}_l$$

## Link relations

$$\underline{u}_l = \underline{L}_{lr} \underline{u}_r$$

$$\bar{\tilde{f}}_l^T \delta \underline{u}_l + \bar{\tilde{f}}_r^T \delta \underline{u}_r = 0 \quad \forall \quad \{\delta \underline{u}_l, \delta \underline{u}_r\} \quad \rightarrow$$

$$\bar{\tilde{f}}_l^T \underline{L}_{lr} + \bar{\tilde{f}}_r^T = \underline{0}^T \quad \rightarrow \quad \underline{L}_{lr}^T \bar{\tilde{f}}_l + \bar{\tilde{f}}_r = \underline{0} \quad \rightarrow \quad \bar{\tilde{f}}_r = -\underline{L}_{lr}^T \bar{\tilde{f}}_l = -\underline{L}_{rl} \bar{\tilde{f}}_l$$

# Partitioning for links

substitution of link relations  $\rightarrow$

$$\left. \begin{array}{l} \underline{K}_{ff} \underline{u}_f + (\underline{K}_{fr} + \underline{K}_{fl} \underline{L}_{lr}) \underline{u}_r = \underline{f}_f \\ \underline{K}_{rf} \underline{u}_f + (\underline{K}_{rr} + \underline{K}_{rl} \underline{L}_{lr}) \underline{u}_r = \underline{f}_r - \underline{L}_{rl} \bar{\underline{f}}_l \\ \underline{K}_{lf} \underline{u}_f + (\underline{K}_{lr} + \underline{K}_{ll} \underline{L}_{lr}) \underline{u}_r = \underline{f}_l + \bar{\underline{f}}_l \end{array} \right\} \rightarrow$$

elimination of  $\bar{\underline{f}}_l$

$$\left. \begin{array}{l} \underline{K}_{ff} \underline{u}_f + (\underline{K}_{fr} + \underline{K}_{fl} \underline{L}_{lr}) \underline{u}_r = \underline{f}_f \\ (\underline{K}_{rf} + \underline{L}_{rl} \underline{K}_{lf}) \underline{u}_f + \\ (\underline{K}_{rr} + \underline{K}_{rl} \underline{L}_{lr} + \underline{L}_{rl} \underline{K}_{lr} + \underline{L}_{rl} \underline{K}_{ll} \underline{L}_{lr}) \underline{u}_r = \underline{f}_r + \underline{L}_{rl} \underline{f}_l \end{array} \right\} \rightarrow$$

$$\left[ \begin{array}{cc} \underline{K}_{ff} & \underline{K}_{fr} + \underline{K}_{fl} \underline{L}_{lr} \\ \underline{K}_{rf} + \underline{L}_{rl} \underline{K}_{lf} & \underline{K}_{rr} + \underline{K}_{rl} \underline{L}_{lr} + \underline{L}_{rl} \underline{K}_{lr} + \underline{L}_{rl} \underline{K}_{ll} \underline{L}_{lr} \end{array} \right] \left[ \begin{array}{c} \underline{u}_f \\ \underline{u}_r \end{array} \right] = \left[ \begin{array}{c} \underline{f}_f \\ \underline{f}_r + \underline{L}_{rl} \underline{f}_l \end{array} \right] \rightarrow$$

$$\underline{K} \underline{u} = \underline{f}$$

# Program structure

```
read input data from input file
calculate additional variables from input data
initialize values and arrays

for all elements
    calculate initial element stiffness matrix
    assemble global stiffness matrix
end element loop

determine external load from input

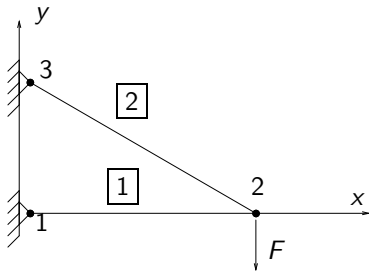
take tyings into account
take boundary conditions into account

calculate nodal displacements

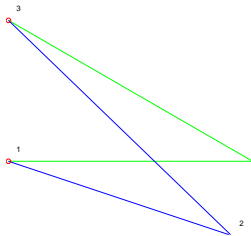
for all elements
    calculate stresses from material behavior
    calculate element internal nodal forces
    assemble global internal load column
end element loop

store data for post-processing
```

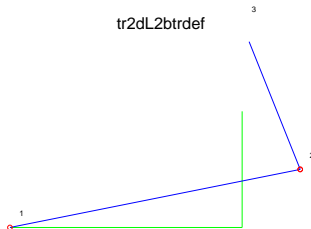
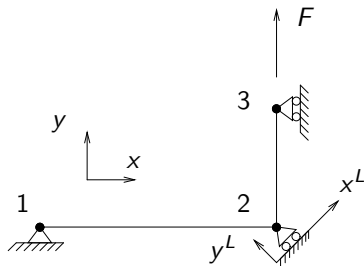
# Simple two-dimensional truss structure



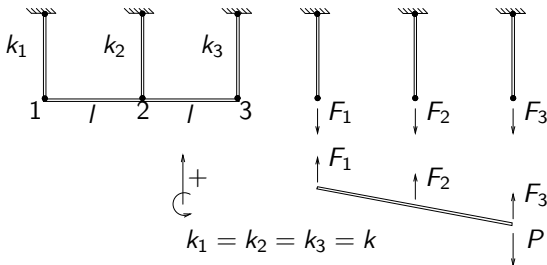
tr2dL2bardef



# Transformation of nodal coordinate system



# Tyings



truss stiffness

equilibrium

$$F_1 + F_2 + F_3 - P = 0 \quad ; \quad -F_1 2l - F_2 l = 0$$

deformation

$$v_1 = -\frac{F_1}{k} \quad ; \quad v_2 = -\frac{F_2}{k} \quad ; \quad v_3 = -\frac{F_3}{k}$$

equilibrium equations in displacements

$$-kv_1 - kv_2 - kv_3 - P = 0 \quad ; \quad 2lkv_1 + lkv_2 = 0$$



## Example : link relations

link relation

$$v_2 = \frac{1}{2} (v_1 + v_3) \quad \rightarrow \quad v_2 = \begin{bmatrix} \frac{1}{2} & \frac{1}{2} \end{bmatrix} \begin{bmatrix} v_1 \\ v_3 \end{bmatrix}$$

elimination of  $v_2$   $\rightarrow$  equation for retained displacements

$$\left. \begin{aligned} -\frac{3}{2}kv_1 - \frac{3}{2}kv_3 - P &= 0 \\ \frac{5}{2}lv_1 + \frac{1}{2}lv_3 &= 0 \end{aligned} \right\} \rightarrow v_1 = -\frac{1}{5}v_3$$

solving

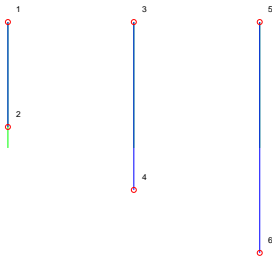
$$\frac{3}{10}kv_3 - \frac{3}{2}kv_3 - P = 0 \quad \rightarrow \quad -\frac{6}{5}kv_3 - P = 0 \quad \rightarrow$$

$$v_3 = -\frac{5}{6}\frac{P}{k} \quad \rightarrow \quad v_1 = \frac{1}{6}\frac{P}{k}$$

$$\text{link} \quad \rightarrow \quad v_2 = -\frac{1}{3}\frac{P}{k}$$

# FE solution

tr2dL3bInkdef



# NONLINEAR DEFORMATION

[back to index](#)

# Strains for large elongation

linear strain

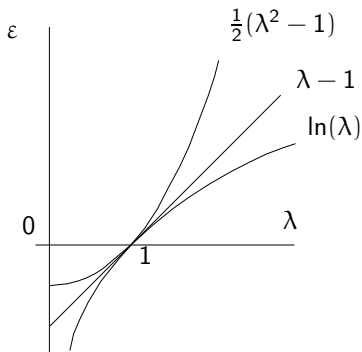
$$\varepsilon = \varepsilon_l = \lambda - 1$$

logarithmic strain

$$\varepsilon = \varepsilon_{ln} = \ln(\lambda)$$

Green-Lagrange strain

$$\varepsilon = \varepsilon_{gl} = \frac{1}{2}(\lambda^2 - 1)$$



# Linear strain

linear strain

$$\varepsilon = \varepsilon_l = \lambda - 1 = \frac{\Delta l}{l_0}$$

contraction strain

$$\varepsilon_d = \mu - 1 = -\nu \varepsilon_l = -\nu(\lambda - 1)$$

change of cross-sectional area

$$\mu = \sqrt{\frac{A}{A_0}} = 1 - \nu(\lambda - 1) \quad \rightarrow \quad A = A_0 \{1 - \nu(\lambda - 1)\}^2$$

restriction of elongation

$$1 - \nu(\lambda - 1) > 0 \quad \rightarrow \quad \lambda - 1 < \frac{1}{\nu} \quad \rightarrow \quad \lambda < \frac{1 + \nu}{\nu}$$

# Logarithmic strain

logarithmic strain

$$\varepsilon = \varepsilon_{ln} = \ln(\lambda)$$

contraction strain

$$\varepsilon_d = \ln(\mu) = -\nu \varepsilon_{ln} = -\nu \ln \lambda$$

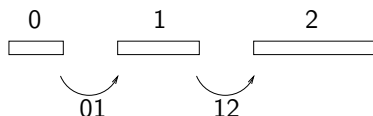
change of cross-sectional area

$$\mu = \sqrt{\frac{A}{A_0}} = e^{-\nu \varepsilon_{ln}} = e^{-\nu \ln(\lambda)} = \left[ e^{\ln(\lambda)} \right]^{-\nu} = \lambda^{-\nu} \quad \rightarrow$$

$$A = A_0 \lambda^{-2\nu}$$

$$\text{NB} \quad : \quad \ln(x) = {}^e \log(x) = y \quad \rightarrow \quad x = e^y$$

# Advantage logarithmic strain



$$l_0 \rightarrow l_1 \quad \begin{aligned} \varepsilon_I(01) &= \frac{l_1 - l_0}{l_0} \\ \varepsilon_{ln}(01) &= \ln\left(\frac{l_1}{l_0}\right) \end{aligned}$$

$$l_1 \rightarrow l_2 \quad \begin{aligned} \varepsilon_I(12) &= \frac{l_2 - l_1}{l_1} \\ \varepsilon_{ln}(12) &= \ln\left(\frac{l_2}{l_1}\right) \end{aligned}$$

$$l_0 \rightarrow l_2 \quad \begin{aligned} \varepsilon_I(02) &= \frac{l_2 - l_0}{l_0} \neq \varepsilon_I(01) + \varepsilon_I(12) \\ \varepsilon_{ln}(02) &= \ln\left(\frac{l_2}{l_0}\right) = \ln\left(\frac{l_2}{l_1} \frac{l_1}{l_0}\right) = \varepsilon_{ln}(01) + \varepsilon_{ln}(12) \end{aligned}$$

# Green-Lagrange strain

Green-Lagrange strain  $\varepsilon = \varepsilon_{gl} = \frac{1}{2}(\lambda^2 - 1)$

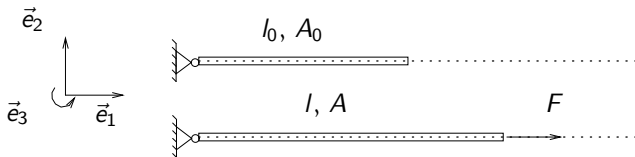
contraction strain  $\varepsilon_d = \frac{1}{2}(\mu^2 - 1) = -\nu \varepsilon_{ln} = -\nu \frac{1}{2}(\lambda^2 - 1)$

change of cross-sectional area

$$1 - \nu(\lambda^2 - 1) > 0 \quad \rightarrow \quad \lambda < \sqrt{\frac{1 + \nu}{\nu}}$$



# Mechanical power for an axially loaded truss

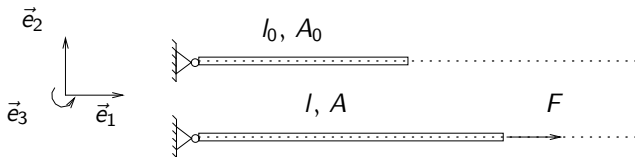


mechanical power

$$P = F\dot{l}$$

$$\begin{array}{lll} \varepsilon_l = \lambda - 1 & \rightarrow & \dot{\varepsilon}_l = \dot{\lambda} = \frac{\dot{l}}{l_0} \\ \varepsilon_{ln} = \ln(\lambda) & \rightarrow & \dot{\varepsilon}_{ln} = \dot{\lambda}\lambda^{-1} = \frac{\dot{l}}{l} \\ \varepsilon_{gl} = \frac{1}{2}(\lambda^2 - 1) & \rightarrow & \dot{\varepsilon}_{gl} = \dot{\lambda}\lambda = \lambda \frac{\dot{l}}{l_0} = \lambda^2 \frac{\dot{l}}{l} \end{array}$$

# Mechanical power for an axially loaded truss



$$P = F\dot{\ell} = F\ell_0\dot{\epsilon}_I = \frac{F}{A_0}A_0\ell_0\dot{\epsilon}_I = \frac{F}{A_0}V_0\dot{\epsilon}_I$$

$$P = F\dot{\ell} = F\ell\dot{\epsilon}_{In} = \frac{F}{A}A\ell\dot{\epsilon}_{In} = \frac{F}{A}V\dot{\epsilon}_{In}$$

$$P = F\dot{\ell} = F\ell_0\dot{\epsilon}_I = \frac{F}{A}A\ell\frac{\ell_0}{\ell}\dot{\epsilon}_I = \frac{F}{A}V\lambda^{-1}\dot{\epsilon}_I$$

$$P = F\dot{\ell} = F\ell\lambda^{-2}\dot{\epsilon}_{gI} = \frac{F}{A}A\ell\lambda^{-2}\dot{\epsilon}_{gI} = \frac{F}{A}V\lambda^{-2}\dot{\epsilon}_{gI}$$

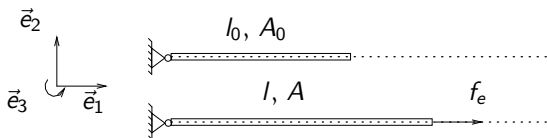
# Mechanical power : stress $\sim$ strain

$$\begin{aligned}
 P &= &= &= V_0 \sigma_n \dot{\epsilon}_I \\
 P &= V \sigma \dot{\epsilon}_{In} &= V_0 (J \sigma) \dot{\epsilon}_{In} &= V_0 \sigma_\kappa \dot{\epsilon}_{In} \\
 P &= V (\sigma \lambda^{-1}) \dot{\epsilon}_I &= V_0 (J \sigma \lambda^{-1}) \dot{\epsilon}_I &= V_0 \sigma_{p1} \dot{\epsilon}_I \\
 P &= V (\sigma \lambda^{-2}) \dot{\epsilon}_{gI} &= V_0 (J \sigma \lambda^{-2}) \dot{\epsilon}_{gI} &= V_0 \sigma_{p2} \dot{\epsilon}_{gI}
 \end{aligned}$$

specific mechanical power :  $P = V_0 \dot{W}_0 = V \dot{W}$

$$\begin{aligned}
 \dot{W}_0 &= \sigma_n \dot{\epsilon}_I &= \sigma_\kappa \dot{\epsilon}_{In} &= \sigma_{p1} \dot{\epsilon}_I &= \sigma_{p2} \dot{\epsilon}_{gI} \\
 \dot{W} &= &= \sigma \dot{\epsilon}_{In} &= \sigma \lambda^{-1} \dot{\epsilon}_I &= \sigma \lambda^{-2} \dot{\epsilon}_{gI}
 \end{aligned}$$

# Equilibrium : linear



external force

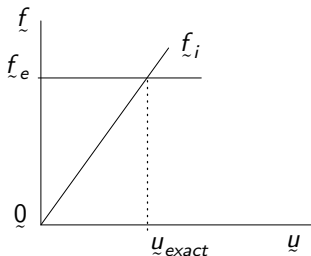
$f_e$

internal force

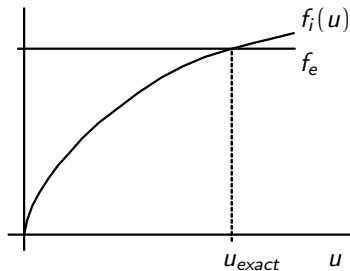
$$f_i = \sigma_n A_0 = E \varepsilon_l A_0 = EA_0 \frac{u}{l_0}$$

equilibrium of point  $P$

$$f_i = \frac{EA_0}{l_0} u = f_e \rightarrow u_{\text{exact}} = \frac{l_0}{EA_0} f_e$$



# Equilibrium : nonlinear



external force

$$f_e$$

internal force

$$f_i = \sigma A = f_i(u)$$

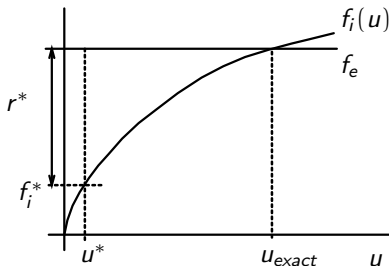
equilibrium of point  $P$

$$f_i(u) = f_e$$

$f_i(u)$  non-linear

iterative solution process needed

# Iterative solution procedure



analytic solution

$$f_i(u_{exact}) = f_e \rightarrow f_e - f_i(u_{exact}) = 0$$

approximation  $u^*$

$$f_e - f_i(u^*) = r(u^*) \neq 0$$

residual

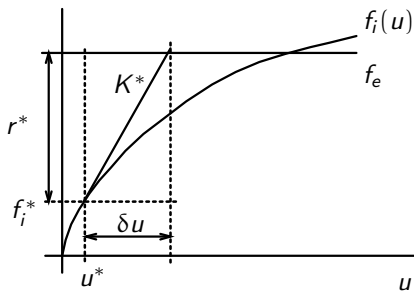
$$r^* = r(u^*)$$

# Newton-Raphson iteration procedure

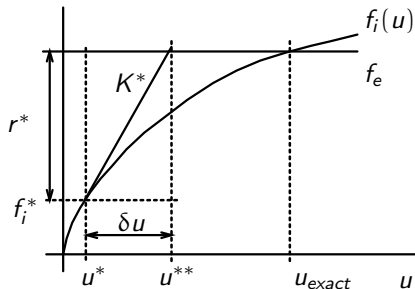
$$\left. \begin{array}{l} f_i(u_{\text{exact}}) = f_e \\ u_{\text{exact}} = u^* + \delta u \end{array} \right\} \rightarrow f_i(u^* + \delta u) = f_e$$

$$f_i(u^*) + \left. \frac{df_i}{du} \right|_{u^*} \delta u = f_e \rightarrow f_i^* + K^* \delta u = f_e$$

$$K^* \delta u = f_e - f_i^* = r^* \rightarrow \delta u = \frac{1}{K^*} r^*$$



# New approximate solution



new approximation

$$u^{**} = u^* + \delta u$$

error

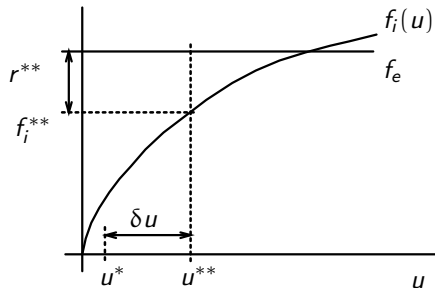
$$u_{exact} - u^{**}$$

error smaller

→ convergence



# Convergence control



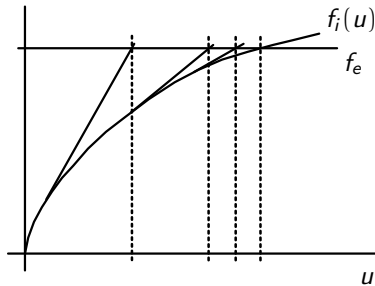
residual force

$$|r^{**}| \leq c_r \rightarrow \text{stop iteration}$$

iterative displacement

$$|\delta u| \leq c_u \rightarrow \text{stop iteration}$$

# Convergence



# Residual and tangential stiffness

internal nodal force

$$f_i^* = N(\lambda^*) = A^* \sigma^*$$

tangential stiffness

$$K^* = \left. \frac{\partial f_i}{\partial u} \right|_{u^*} = \left. \frac{\partial N(\lambda)}{\partial u} \right|_{u^*} = \left. \frac{dN}{d\lambda} \right|_{\lambda^*} \frac{d\lambda}{du}$$

geometry

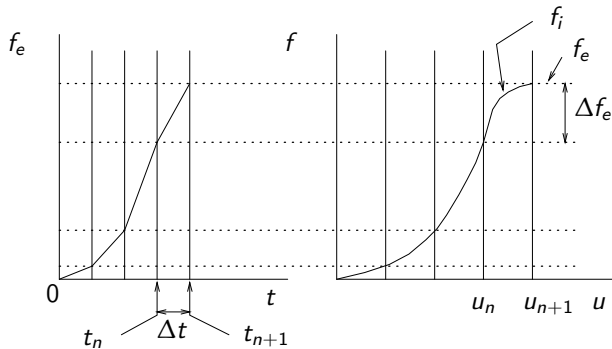
$$\lambda = 1 + \frac{\Delta l}{l_0} = 1 + \frac{1}{l_0} u \quad \rightarrow \quad \frac{d\lambda}{du} = \frac{1}{l_0}$$

tangential stiffness

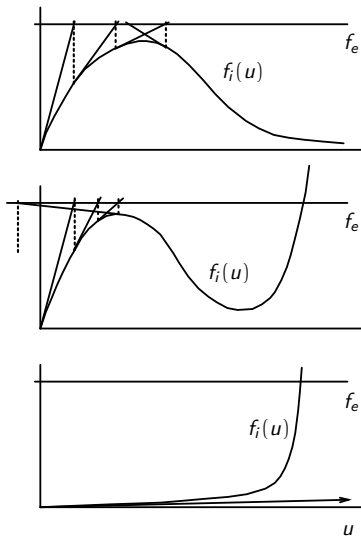
$$K^* = \left. \frac{dN}{d\lambda} \right|_{\lambda^*} \frac{\partial \lambda}{\partial u} = \left. \frac{dN}{d\lambda} \right|_{\lambda^*} \frac{1}{l_0} = \frac{1}{l_0} \left. \frac{d}{d\lambda} (\sigma A) \right|_{\lambda^*}^*$$

$$K^* = \frac{1}{l_0} \left. \frac{d\sigma}{d\lambda} \right|_{\lambda^*}^* A^* + \frac{1}{l_0} \sigma^* \left. \frac{dA}{d\lambda} \right|_{\lambda^*}^*$$

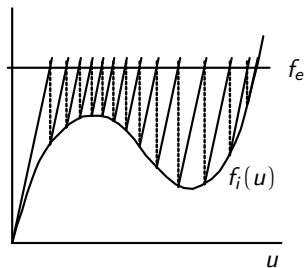
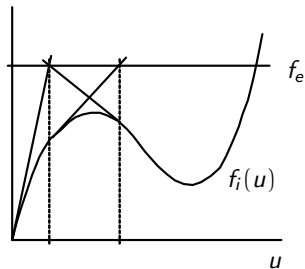
# Incremental loading



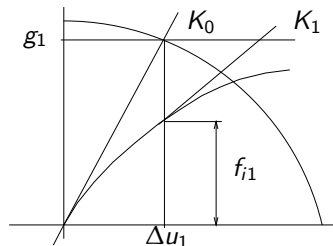
# Non-converging solution process



# Modified Newton-Raphson procedure

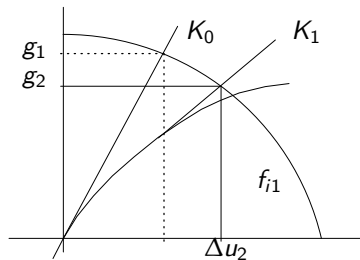


# Path-following solution algorithm



$$K_0 \delta u_1 = f_{e0} + \lambda_1 f_{ef} = g_1 \quad \rightarrow \quad \delta u_1 = K_0^{-1} g_1 \quad \rightarrow$$
$$\Delta u_1 = \delta u_1 \quad ; \quad u_1 = u_0 + \Delta u_1 \quad \rightarrow \quad f_{i1} , K_1$$

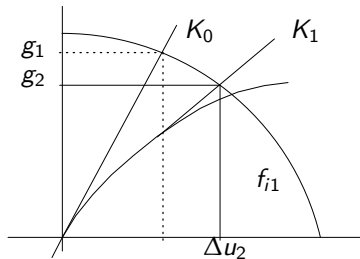
# Path-following solution algorithm



$$\begin{aligned} K_1 \delta u_2 &= g_2 - f_{i2} = f_{e0} + \lambda_2 f_{ef} - f_{i2} \quad \rightarrow \\ \delta u_2 &= K_1^{-1} (f_{e0} + \lambda_2 f_{ef} - f_{i2}) \end{aligned}$$



# Path-following solution algorithm



$$\begin{bmatrix} \Delta u_2 & g_2 \end{bmatrix} \begin{bmatrix} \Delta u_2 \\ g_2 \end{bmatrix} = (\Delta u_2)^2 + (g_2)^2 = C^2 \rightarrow$$

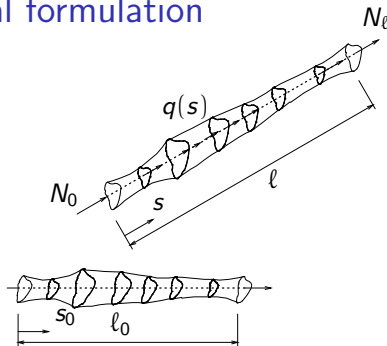
$$(\Delta u_1 + K_1^{-1} f_{e0} + K_1^{-1} f_{ef} \lambda_2 - K_1^{-1} f_{i2})^2 + (f_{e0} + \lambda_2 f_{ef})^2 = C^2 \rightarrow$$

$$\lambda_2 \rightarrow \delta u_2 \rightarrow \Delta u_2, u_2 \rightarrow f_{i3}, K_2$$

## WEIGHTED RESIDUAL FORMULATION

[back to index](#)

# Weighted residual formulation



equilibrium  $\frac{d\vec{N}}{ds} + \vec{q}(s) = \vec{0} \rightarrow \frac{d(\sigma A \vec{n})}{ds} + \vec{q}(s) = \vec{0} \quad \forall s \in [0, \ell]$

approximation  $\frac{d(\sigma^* A^* \vec{n})}{ds} + \vec{q}(s) = \vec{\Delta}(s) \neq \vec{0} \quad \forall s \in [0, \ell]$

weighted error  $\vec{\Delta}(s)$  is "smeared out" over  $[0, \ell] \rightarrow \int_{s=0}^{s=\ell} \vec{w}(s) \cdot \vec{\Delta}(s) ds$

# Weighted residual formulation

$$\int_{s=0}^{s=\ell} \vec{w} \cdot \left\{ \frac{d(\sigma A \vec{n})}{ds} + \vec{q} \right\} ds = 0 \quad \forall \quad \vec{w}(s)$$

partial integration of 1st term  $\rightarrow$  weak formulation

$$\begin{aligned} \int_{s=0}^{s=\ell} \frac{d\vec{w}}{ds} \cdot (\sigma A \vec{n}) ds &= \int_{s=0}^{s=\ell} \vec{w} \cdot \vec{q} ds + \left[ \vec{w}(\ell) \cdot \vec{N}(\ell) - \vec{w}(0) \cdot \vec{N}(0) \right] \\ &= f_e(\vec{w}) \quad \forall \quad \vec{w}(s) \end{aligned}$$

# State transformation

$$\int_{s=0}^{s=\ell} \frac{d\vec{w}}{ds} \cdot (\sigma A \vec{n}) ds = f_e(\vec{w}) \quad \forall \quad \vec{w}(s)$$

$$\frac{d(\quad)}{ds} = \frac{ds_0}{ds} \frac{d(\quad)}{ds_0} = \frac{1}{\lambda} \frac{d(\quad)}{ds_0} \quad ; \quad ds = \lambda ds_0$$

integral transformation

$$\int_{s_0=0}^{s_0=\ell_0} \frac{d\vec{w}}{ds_0} \cdot (\sigma A \vec{n}) ds_0 = f_{e0}(\vec{w}) \quad \forall \quad \vec{w}(s_0)$$

## Iterative solution process

$$\int_{s_0=0}^{s_0=\ell_0} \frac{d\vec{w}}{ds_0} \cdot (\sigma^* + \delta\sigma)(A^* + \delta A)(\vec{n}^* + \delta\vec{n}) ds_0 = f_{e0}(\vec{w}) \quad \forall \quad \vec{w}(s_0)$$

# Linearization

$$\int_{s_0=0}^{s_0=\ell_0} \frac{d\vec{w}}{ds_0} \cdot (\sigma^* + \delta\sigma)(A^* + \delta A)(\vec{n}^* + \delta\vec{n}) ds_0 = f_{e0}(\vec{w}) \quad \forall \quad \vec{w}(s_0)$$

$$\begin{aligned} \int_{s_0=0}^{s_0=\ell_0} \frac{d\vec{w}}{ds_0} \cdot \delta\sigma A^* \vec{n}^* ds_0 + \int_{s_0=0}^{s_0=\ell_0} \frac{d\vec{w}}{ds_0} \cdot \sigma^* A^* \delta\vec{n} ds_0 \\ = f_{e0}(\vec{w}) - \int_{s_0=0}^{s_0=\ell_0} \frac{d\vec{w}}{ds_0} \cdot \sigma^* A^* \vec{n}^* ds_0 \quad \forall \quad \vec{w}(s_0) \end{aligned}$$

## Material model $\rightarrow$ iterative stress change

$$\int_{s_0=0}^{s_0=\ell_0} \frac{d\vec{w}}{ds_0} \cdot (\sigma^* + \delta\sigma)(A^* + \delta A)(\vec{n}^* + \delta\vec{n}) ds_0 = f_{e0}(\vec{w}) \quad \forall \quad \vec{w}(s_0)$$

$$\begin{aligned} \int_{s_0=0}^{s_0=\ell_0} \frac{d\vec{w}}{ds_0} \cdot \delta\sigma A^* \vec{n}^* ds_0 + \int_{s_0=0}^{s_0=\ell_0} \frac{d\vec{w}}{ds_0} \cdot \sigma^* A^* \delta\vec{n} ds_0 \\ = f_{e0}(\vec{w}) - \int_{s_0=0}^{s_0=\ell_0} \frac{d\vec{w}}{ds_0} \cdot \sigma^* A^* \vec{n}^* ds_0 \quad \forall \quad \vec{w}(s_0) \end{aligned}$$

$$\sigma = \sigma(\lambda) \quad \rightarrow \quad \delta\sigma = \left. \frac{d\sigma}{d\lambda} \right|^* \delta\lambda = \left. \frac{d\sigma}{d\lambda} \right|^* \frac{d(\delta s)}{ds_0} = \left. \frac{d\sigma}{d\lambda} \right|^* \vec{n}^* \cdot \frac{d(\delta \vec{u})}{ds_0}$$



## Rotation → iterative orientation change

$$\vec{n} = \frac{d\vec{x}}{ds} = \frac{ds_0}{ds} \frac{d\vec{x}}{ds_0} = \frac{1}{\lambda} \frac{d\vec{x}}{ds_0}$$

$$\begin{aligned}\delta\vec{n} &= \left[ -\frac{1}{\lambda^2} \frac{d\vec{x}}{ds_0} \right]^* \delta\lambda + \left[ \frac{1}{\lambda} \right]^* \frac{d(\delta\vec{x})}{ds_0} = \left[ -\frac{1}{\lambda} \vec{n} \right]^* \delta\lambda + \left[ \frac{1}{\lambda} \right]^* \frac{d(\delta\vec{x})}{ds_0} \\ &= \left[ -\frac{1}{\lambda} \vec{n}\vec{n} \right]^* \cdot \frac{d(\delta\vec{u})}{ds_0} + \left[ \frac{1}{\lambda} \right]^* \frac{d(\delta\vec{u})}{ds_0} = \left[ (\mathbf{I} - \vec{n}\vec{n}) \frac{1}{\lambda} \right]^* \cdot \frac{d(\delta\vec{u})}{ds_0} \\ &= \left[ \vec{n}\vec{n} \frac{1}{\lambda} \right]^* \cdot \frac{d(\delta\vec{u})}{ds_0}\end{aligned}$$

# Iterative weighted residual integral

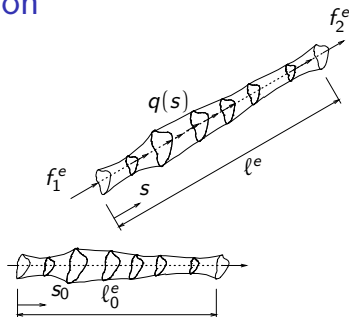
$$\begin{aligned}
 & \int_{s_0=0}^{s_0=\ell_0} \frac{d\vec{w}}{ds_0} \cdot \left( \left. \frac{d\sigma}{d\lambda} \right|^* \vec{n}^* \cdot \frac{d(\delta\vec{u})}{ds_0} \right) A^* \vec{n}^* ds_0 + \\
 & \quad \int_{s_0=0}^{s_0=\ell_0} \frac{d\vec{w}}{ds_0} \cdot \sigma^* A^* \left( \vec{m}^* \vec{m}^* \cdot \frac{1}{\lambda^*} \frac{d(\delta\vec{u})}{ds_0} \right) ds_0 \\
 & = f_{e0}(\vec{w}) - \int_{s_0=0}^{s_0=\ell_0} \frac{d\vec{w}}{ds_0} \cdot \sigma^* A^* \vec{n}^* ds_0 \quad \forall \quad \vec{w}(s_0)
 \end{aligned}$$

$$\begin{aligned}
 & \int_{s_0=0}^{s_0=\ell_0} \frac{d\vec{w}}{ds_0} \cdot \vec{n}^* \left( \left. \frac{d\sigma}{d\lambda} \right|^* A^* \right) \vec{n}^* \cdot \frac{d(\delta\vec{u})}{ds_0} ds_0 + \\
 & \quad \int_{s_0=0}^{s_0=\ell_0} \frac{d\vec{w}}{ds_0} \cdot \vec{m}^* \left( \sigma^* A^* \frac{1}{\lambda^*} \right) \vec{m}^* \cdot \frac{d(\delta\vec{u})}{ds_0} ds_0 \\
 & = f_{e0}(\vec{w}) - \int_{s_0=0}^{s_0=\ell_0} \frac{d\vec{w}}{ds_0} \cdot \sigma^* A^* \vec{n}^* ds_0 \quad \forall \quad \vec{w}(s_0)
 \end{aligned}$$

# FINITE ELEMENT METHOD

[back to index](#)

# Element equation



local coordinate :  $-1 \leq \xi \leq 1$  ;  $ds_0 = \frac{l_0}{2} d\xi$  ;  $\frac{d(\cdot)}{ds_0} = \frac{2}{l_0} \frac{d(\cdot)}{d\xi}$

$$\int_{\xi=-1}^{\xi=1} \frac{d\vec{w}}{d\xi} \cdot \vec{n}^* \left( \frac{d\sigma}{d\lambda} \right)^* A^* \frac{2}{l_0} \vec{n}^* \cdot \frac{d(\delta\vec{u})}{d\xi} d\xi +$$

$$\int_{\xi=-1}^{\xi=1} \frac{d\vec{w}}{d\xi} \cdot \vec{m}^* \left( \sigma^* A^* \frac{1}{\lambda^*} \frac{2}{l_0} \right) \vec{m}^* \cdot \frac{d(\delta\vec{u})}{d\xi} d\xi = f_{e0}^e(\vec{w}) - \int_{\xi=-1}^{\xi=1} \frac{d\vec{w}}{d\xi} \cdot \sigma^* A^* \vec{n}^* d\xi$$

# Components

$$\begin{aligned}
 & \int_{\xi=-1}^{\xi=1} \frac{d\tilde{w}^T}{d\xi} \tilde{n}^* \left( \left. \frac{d\sigma}{d\lambda} \right|^* A^* \frac{2}{l_0} \right) \tilde{n}^{*T} \frac{d(\delta\tilde{u})}{d\xi} d\xi + \\
 & \int_{\xi=-1}^{\xi=1} \frac{d\tilde{w}^T}{d\xi} \tilde{m}^* \left( \sigma^* A^* \frac{1}{\lambda^*} \frac{2}{l_0} \right) \tilde{m}^{*T} \frac{d(\delta\tilde{u})}{d\xi} d\xi, \\
 & = f_{e0}^e(\tilde{w}) - \int_{\xi=-1}^{\xi=1} \frac{d\tilde{w}^T}{d\xi} \sigma^* A^* \tilde{n}^* d\xi
 \end{aligned}$$

# Interpolation

$$\delta \underline{u}^T = \begin{bmatrix} \delta u_1 & \delta u_2 \end{bmatrix} = \begin{bmatrix} \delta u_{11}\psi^1 + \delta u_{21}\psi^2 & \delta u_{12}\psi^1 + \delta u_{22}\psi^2 \end{bmatrix}$$

$$\underline{w}^T = \begin{bmatrix} w_{11}\psi^1 + w_{21}\psi^2 & w_{12}\psi^1 + w_{22}\psi^2 \end{bmatrix}$$

$$\text{with} \quad \psi^1(\xi) = \frac{1}{2}(1 - \xi) \quad ; \quad \psi^2(\xi) = \frac{1}{2}(1 + \xi)$$

$$\begin{aligned} \frac{d(\delta \underline{u})}{d\xi} &= \begin{bmatrix} \frac{d(\delta u_1)}{d\xi} \\ \frac{d(\delta u_2)}{d\xi} \end{bmatrix} = \begin{bmatrix} \frac{d\psi^1}{d\xi} & 0 & \frac{d\psi^2}{d\xi} & 0 \\ 0 & \frac{d\psi^1}{d\xi} & 0 & \frac{d\psi^2}{d\xi} \end{bmatrix} \begin{bmatrix} \delta u_{11} \\ \delta u_{12} \\ \delta u_{21} \\ \delta u_{22} \end{bmatrix} \\ &= \frac{1}{2} \begin{bmatrix} -1 & 0 & 1 & 0 \\ 0 & -1 & 0 & 1 \end{bmatrix} \delta \underline{u}^e \end{aligned}$$

$$\begin{aligned} \frac{d\underline{w}^T}{d\xi} &= \begin{bmatrix} \frac{dw_1}{d\xi} & \frac{dw_2}{d\xi} \end{bmatrix} = \begin{bmatrix} w_{11} & w_{12} & w_{21} & w_{22} \end{bmatrix} \begin{bmatrix} \frac{d\psi^1}{d\xi} & 0 \\ 0 & \frac{d\psi^1}{d\xi} \\ \frac{d\psi^2}{d\xi} & 0 \\ 0 & \frac{d\psi^2}{d\xi} \end{bmatrix} \\ &= \underline{w}^{eT} \frac{1}{2} \begin{bmatrix} -1 & 0 \\ 0 & -1 \\ 1 & 0 \\ 0 & 1 \end{bmatrix} \end{aligned}$$

# Element equation

$$\begin{aligned}
 & \tilde{w}^{eT} \int_{\xi=-1}^{\xi=1} \begin{bmatrix} -1 & 0 \\ 0 & -1 \\ 1 & 0 \\ 0 & 1 \end{bmatrix} \begin{bmatrix} c \\ s \end{bmatrix}^* \frac{1}{4} \left( \frac{d\sigma}{d\lambda} \right)^* A^* \frac{2}{l_0} \\
 & \quad \begin{bmatrix} c & s \end{bmatrix}^* \begin{bmatrix} -1 & 0 & 1 & 0 \\ 0 & -1 & 0 & 1 \end{bmatrix} d\xi \delta \underline{u}^e + \\
 & \tilde{w}^{eT} \int_{\xi=-1}^{\xi=1} \begin{bmatrix} -1 & 0 \\ 0 & -1 \\ 1 & 0 \\ 0 & 1 \end{bmatrix} \begin{bmatrix} -s \\ c \end{bmatrix}^* \frac{1}{4} \left( \sigma^* A^* \frac{1}{\lambda^*} \frac{2}{l_0} \right) \\
 & \quad \begin{bmatrix} -s & c \end{bmatrix}^* \begin{bmatrix} -1 & 0 & 1 & 0 \\ 0 & -1 & 0 & 1 \end{bmatrix} d\xi \delta \underline{u}^e \\
 & = f_{e0}^e(\tilde{w}^e) - \tilde{w}^{eT} \int_{\xi=-1}^{\xi=1} \begin{bmatrix} -1 & 0 \\ 0 & -1 \\ 1 & 0 \\ 0 & 1 \end{bmatrix} \frac{1}{2} \begin{bmatrix} c \\ s \end{bmatrix}^* (\sigma^* A^*) d\xi
 \end{aligned}$$

# Element equation

$$\begin{aligned}
 & \tilde{w}^{eT} \int_{\xi=-1}^{\xi=1} \left( \frac{1}{2} \frac{d\sigma}{d\lambda} \right)^* A^* \frac{1}{l_0} \begin{bmatrix} -c \\ -s \\ c \\ s \end{bmatrix}^* \begin{bmatrix} -c & -s & c & s \end{bmatrix}^* d\xi \delta \underline{u}^e + \\
 & \tilde{w}^{eT} \int_{\xi=-1}^{\xi=1} \left( \frac{1}{2} \sigma^* A^* \frac{1}{\lambda^*} \frac{1}{l_0} \right) \begin{bmatrix} s \\ -c \\ -s \\ c \end{bmatrix}^* \begin{bmatrix} s & -c & -s & c \end{bmatrix}^* d\xi \delta \underline{u}^e \\
 & = f_{e0}^e(\tilde{w}^e) - \tilde{w}^{eT} \int_{\xi=-1}^{\xi=1} \frac{1}{2} (\sigma^* A^*) \begin{bmatrix} -c \\ -s \\ c \\ s \end{bmatrix}^* d\xi
 \end{aligned}$$



# Element equation

$$\begin{aligned}
 & \tilde{w}^{eT} \int_{\xi=-1}^{\xi=1} \left( \frac{1}{2} \frac{d\sigma}{d\lambda} \right)^* A^* \frac{1}{l_0} \begin{bmatrix} c^2 & cs & -c^2 & -cs \\ cs & s^2 & -cs & -s^2 \\ -c^2 & -cs & c^2 & cs \\ -cs & -s^2 & cs & s^2 \end{bmatrix}^* d\xi \delta \underline{u}^e + \\
 & \tilde{w}^{eT} \int_{\xi=-1}^{\xi=1} \left( \frac{1}{2} \sigma^* A^* \frac{1}{\lambda^*} \frac{1}{l_0} \right) \begin{bmatrix} s^2 & -cs & -s^2 & cs \\ -cs & c^2 & cs & -c^2 \\ -s^2 & cs & s^2 & -cs \\ cs & -c^2 & -cs & c^2 \end{bmatrix}^* d\xi \delta \underline{u}^e \\
 & = f_{e0}^e(\tilde{w}^e) - \tilde{w}^{eT} \int_{\xi=-1}^{\xi=1} \frac{1}{2} (\sigma^* A^*) \begin{bmatrix} -c \\ -s \\ c \\ s \end{bmatrix}^* d\xi
 \end{aligned}$$

# Element equation

$$\begin{aligned}
 & \underline{\tilde{w}}^{eT} \left[ \int_{\xi=-1}^{\xi=1} \left( \frac{1}{2} \frac{d\sigma}{d\lambda} \right)^* A^* \frac{1}{l_0} d\xi \underline{\underline{M}}_L^* \right] \delta \underline{\underline{u}}^e + \\
 & \quad \underline{\tilde{w}}^{eT} \left[ \int_{\xi=-1}^{\xi=1} \left( \frac{1}{2} \sigma^* A^* \frac{1}{\lambda^*} \frac{1}{l_0} \right) d\xi \underline{\underline{M}}_N^* \right] \delta \underline{\underline{u}}^e \\
 & = f_{e0}^e(\underline{\tilde{w}}^e) - \underline{\tilde{w}}^{eT} \int_{\xi=-1}^{\xi=1} \frac{1}{2} (\sigma^* A^*) \underline{\underline{v}}^* d\xi \\
 \\ 
 & \underline{\tilde{w}}^{eT} \underline{\underline{K}}^{e*} \delta \underline{\underline{u}}^e = \underline{\tilde{w}}^{eT} \underline{\underline{f}}_{e0}^e - \underline{\tilde{w}}^{eT} \underline{\underline{f}}_i^{e*} = \underline{\tilde{w}}^{eT} \underline{\underline{r}}^{e*}
 \end{aligned}$$

# Integration

tangential stiffness matrix

$$\underline{K}^{e*} = \left( \frac{d\sigma}{d\lambda} \right)^* A^* \frac{1}{l_0} \begin{bmatrix} c^2 & cs & -c^2 & -cs \\ cs & s^2 & -cs & -s^2 \\ -c^2 & -cs & c^2 & cs \\ -cs & -s^2 & cs & s^2 \end{bmatrix}^* +$$
$$\left( \sigma^* A^* \frac{1}{l^*} \right) \begin{bmatrix} s^2 & -cs & -s^2 & cs \\ -cs & c^2 & cs & -c^2 \\ -s^2 & cs & s^2 & -cs \\ cs & -c^2 & -cs & c^2 \end{bmatrix}^*$$

internal nodal forces

$$\tilde{f}_i^{e*} = \sigma^* A^* \begin{bmatrix} -c \\ -s \\ c \\ s \end{bmatrix}^*$$

# Assembling

element contribution

$$\underline{w}^{eT} \underline{K}^{e*} \delta \underline{u}^e = \underline{w}^{eT} \underline{f}_{e0}^e - \underline{w}^{eT} \underline{f}_i^{e*} = \underline{w}^{eT} \underline{r}^{e*}$$

assembled equation

$$\underline{w}^T \underline{K}^* \delta \underline{u} = \underline{w}^T \underline{f}_{e0} - \underline{w}^T \underline{f}_i^* = \underline{w}^T \underline{r}^* \quad \forall \underline{w}$$

iterative equation system

$$\underline{K}^* \delta \underline{u} = \underline{r}^*$$

# Program structure

```
read input data from input file
calculate additional variables from input data
initialize values and arrays

while load increments to be done

    for all elements
        calculate initial element stiffness matrix
        assemble global stiffness matrix
    end element loop

    determine external incremental load from input

    while non-converged iteration step

        take tyings into account
        take boundary conditions into account

        calculate iterative nodal displacements
        calculate total deformation

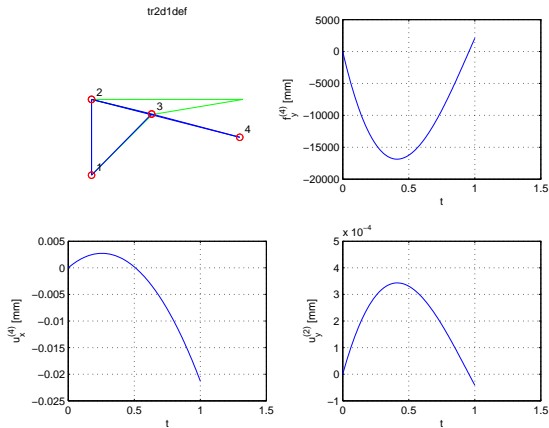
        for all elements
            calculate stresses from material behavior
            calculate material stiffness from material behavior
            calculate element internal nodal forces
            calculate element stiffness matrix
            assemble global stiffness matrix
            assemble global internal load column
        end element loop

        calculate residual load column
        calculate convergence norm

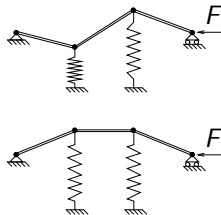
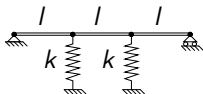
    end iteration step

    store data for post-processing
end load increment
```

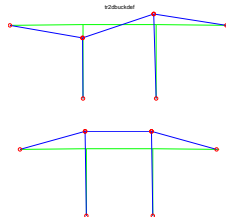
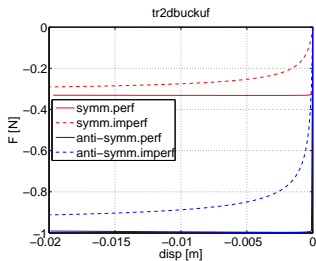
# Large deformation of a truss structure



# Buckling



$$\text{symm} : F_c = \frac{kl}{3} \quad ; \quad \text{anti-symm} : F_c = kl$$



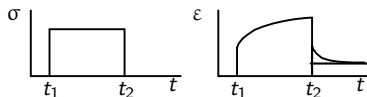
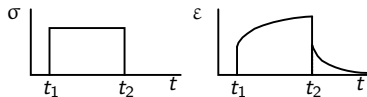
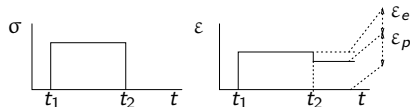
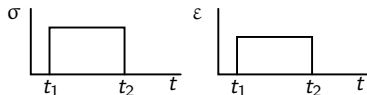
# ONE-DIMENSIONAL MATERIAL BEHAVIOR

[back to index](#)

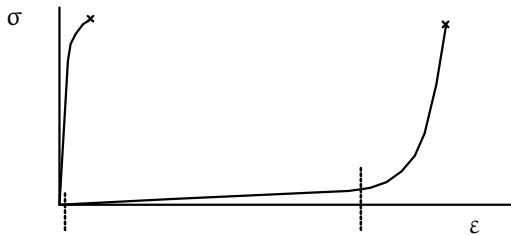
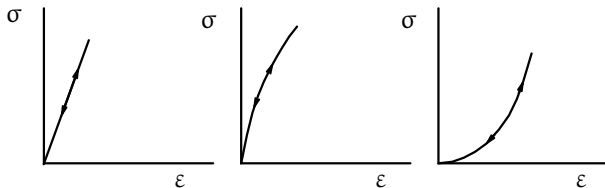


# Time history plots

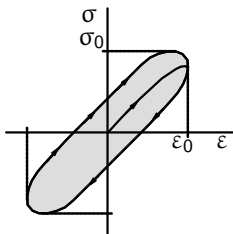
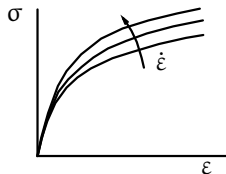
elastic, elastoplastic, viscoelastic, viscoplastic



# Tensile curve : elastic behavior



# Tensile curve : viscoelastic behavior

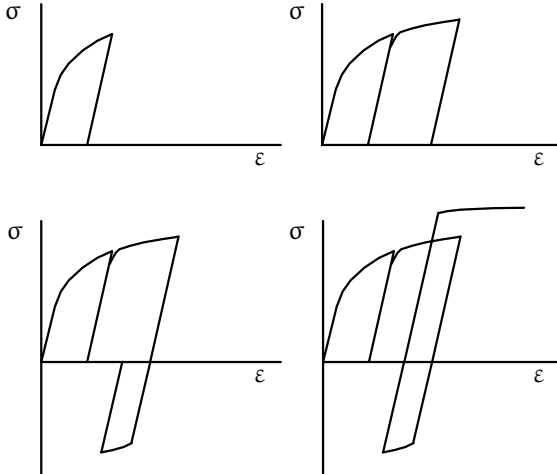


• energy dissipation

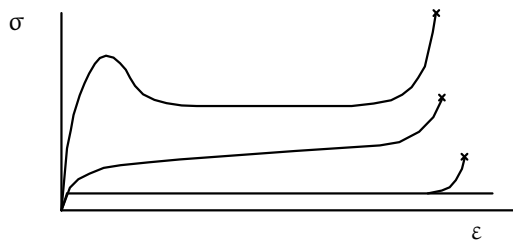
→

heat

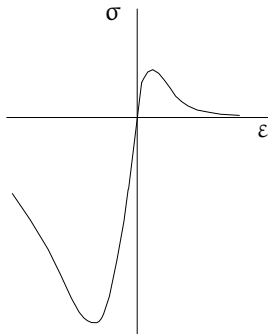
# Tensile curve : elastoplastic behavior



## Tensile curve : viscoplastic behavior



# Tensile curve : damage



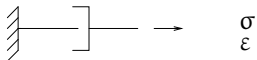
- necking / stable necking
- softening
- fracture
- ductile / brittle

# Discrete material models

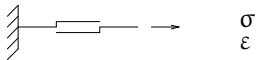
spring



dashpot



friction slider

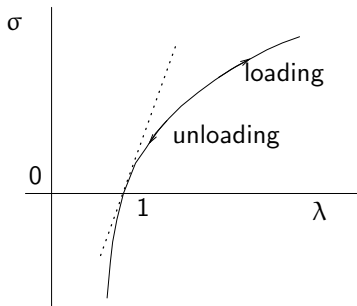


ELASTIC

[back to index](#)



# Elastic material behavior



- no permanent deformation after unloading
- no path- or time dependency
- no energy dissipation

# Small strain elastic behavior

strain

$$\varepsilon = \varepsilon_{gl} = \varepsilon_{ln} = \varepsilon_l = \lambda - 1$$

stress

$$\sigma = \frac{F}{A} = \frac{F}{A_0} = \sigma_n$$

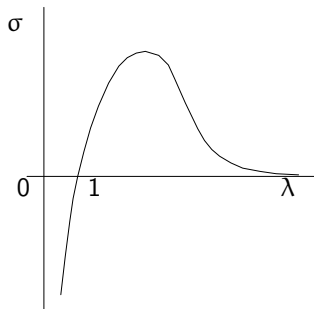
linear elastic behavior

$$\sigma = E\varepsilon = E(\lambda - 1)$$

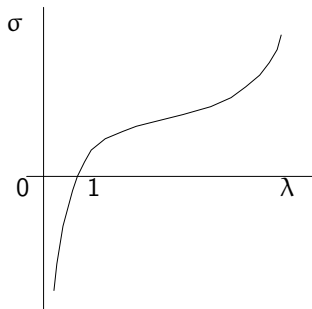
modulus

$$E = \lim_{\lambda \rightarrow 1} \frac{d\sigma}{d\lambda} = \lim_{\varepsilon \rightarrow 0} \frac{d\sigma}{d\varepsilon}$$

# Large strain elastic behavior

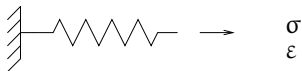


atomic bond



rubber

# Elasticity models



constitutive equation

$$\sigma = \sigma(\lambda)$$

stiffness

$$C_\lambda = \frac{d\sigma}{d\lambda} = \frac{d\sigma}{d\varepsilon} \frac{d\varepsilon}{d\lambda} = C_\varepsilon \frac{d\varepsilon}{d\lambda}$$

elastic models (examples)

$$\left\{ \begin{array}{ll} \text{linear true-log.} & \sigma = C \ln(\lambda) = C \varepsilon_{ln} \\ \text{linear eng.-lin.} & \sigma_n = C(\lambda - 1) = C \varepsilon_I \end{array} \right.$$

# Hyper-elastic models, incompressible

incompressible deformation

$$\frac{\Delta V}{V} = J = \det(\mathbf{F}) = \lambda_1 \lambda_2 \lambda_3 = 1$$

specific energy

$$W = \sum_i^n \sum_j^m C_{ij} (I_1 - 3)^i (I_2 - 3)^j \quad \text{with} \quad C_{00} = 0$$

$$I_1 = \lambda_1^2 + \lambda_2^2 + \lambda_3^2$$

$$I_2 = \lambda_1^2 \lambda_2^2 + \lambda_2^2 \lambda_3^2 + \lambda_3^2 \lambda_1^2 = \frac{1}{\lambda_3^2} + \frac{1}{\lambda_1^2} + \frac{1}{\lambda_2^2}$$

change of specific energy

$$dW = \sigma_1 d\varepsilon_{ln_1} + \sigma_2 d\varepsilon_{ln_2} + \sigma_3 d\varepsilon_{ln_3}$$

# Mooney models

Neo-Hookean  $W = C_{10} (I_1 - 3)$

Mooney-Rivlin  $W = C_{10} (I_1 - 3) + C_{01} (I_2 - 3)$

Signiorini  $W = C_{10}(I_1 - 3) + C_{01}(I_2 - 3) + C_{20}(I_1 - 3)^2$

Yeoh  $W = C_{10}(I_1 - 3) + C_{20}(I_1 - 3)^2 + C_{30}(I_1 - 3)^3$

Klosner-Segal  $W = C_{10}(I_1 - 3) + C_{01}(I_2 - 3) + C_{20}(I_1 - 3)^2 + C_{03}(I_2 - 3)^3$

2-order invariant  $W = C_{10}(I_1 - 3) + C_{01}(I_2 - 3) + C_{11}(I_1 - 3)(I_2 - 3) + C_{20}(I_1 - 3)^2$

Third-order model of James, Green and Simpson

$$\begin{aligned} W = & C_{10}(I_1 - 3) + C_{01}(I_2 - 3) + C_{11}(I_1 - 3)(I_2 - 3) + \\ & C_{20}(I_1 - 3)^2 + C_{02}(I_2 - 3)^2 + C_{21}(I_1 - 3)^2(I_2 - 3) + \\ & C_{30}(I_1 - 3)^3 + C_{03}(I_2 - 3)^3 + C_{12}(I_1 - 3)(I_2 - 3)^2 \end{aligned}$$

# Ogden models

slightly compressible

$$W = \sum_{i=1}^N \frac{a_i}{b_i} \left[ J^{-\frac{b_i}{3}} \left( \lambda_1^{b_i} + \lambda_2^{b_i} + \lambda_3^{b_i} \right) - 3 \right] + 4.5K \left( J^{\frac{1}{3}} - 1 \right)^2$$

$K$  = bulk modulus

$J$  = volume change factor =  $\lambda_1 \lambda_2 \lambda_3$

highly compressible

$$W = \sum_{i=1}^N \frac{a_i}{b_i} \left( \lambda_1^{b_i} + \lambda_2^{b_i} + \lambda_3^{b_i} - 3 \right) + \sum_{i=1}^N \frac{a_i}{c_i} (1 - J^{c_i})$$

## One-dimensional models : Neo-Hookean

$$W = C_{10} \left( \lambda^2 + \frac{2}{\lambda} - 3 \right)$$

$$\sigma = C_{10} \left( 2\lambda - \frac{2}{\lambda^2} \right) \lambda = 2C_{10} \left( \lambda^2 - \frac{1}{\lambda} \right)$$

$$C_\lambda = \frac{\partial \sigma}{\partial \lambda} = 2C_{10} \left( 2\lambda + \frac{1}{\lambda^2} \right)$$

$$E = \lim_{\lambda \rightarrow 1} \frac{\partial \sigma}{\partial \lambda} = 6C_{10}$$

$$F = \sigma A = \sigma \frac{1}{\lambda} A_0 = 2C_{10} A_0 \left( \lambda - \frac{1}{\lambda^2} \right)$$

$$\sigma = \frac{\rho RT}{M} \left( \lambda^2 - \frac{1}{\lambda} \right)$$

with

$\rho$	:	density
$R$	:	gas constant = $8.314 \text{ J K}^{-1} \text{ mol}^{-1}$
$T$	:	absolute temperature
$M$	:	average molecular weight



## One-dimensional models : Mooney-Rivlin

$$W = C_{10} \left( \lambda^2 + \frac{2}{\lambda} - 3 \right) + C_{01} \left( \frac{1}{\lambda^2} + 2\lambda - 3 \right)$$

$$\sigma = 2C_{10} \left( \lambda^2 - \frac{1}{\lambda} \right) + 2C_{01} \left( \lambda^2 - \frac{1}{\lambda} \right) \frac{1}{\lambda}$$

$$C_\lambda = \frac{\partial \sigma}{\partial \lambda} = 2C_{10} \left( 2\lambda + \frac{1}{\lambda^2} \right) + 2C_{01} \left( 1 + \frac{2}{\lambda^3} \right)$$

$$E = \lim_{\lambda \rightarrow 1} \frac{\partial \sigma}{\partial \lambda} = 6(C_{10} + C_{01})$$

$$F = \sigma A = \sigma \frac{1}{\lambda} A_0$$

$$= A_0 \frac{1}{\lambda} \left[ 2C_{10} \left( \lambda^2 - \frac{1}{\lambda} \right) + 2C_{01} \left( \lambda^2 - \frac{1}{\lambda} \right) \frac{1}{\lambda} \right]$$

# NUMERICAL IMPLEMENTATION

[back to index](#)

# Stress update

stress update

$$\sigma = \sigma(\lambda)$$

# Stiffness

stress update

$$\sigma = \sigma(t + \Delta t) = \sigma(\lambda(t + \Delta t)) = \sigma(\lambda)$$

stiffness

$$C_\lambda = \frac{\partial \sigma}{\partial \lambda}$$

# Implementation

`tr2delas.m`

`tr2delam.m`

# Strain excitation

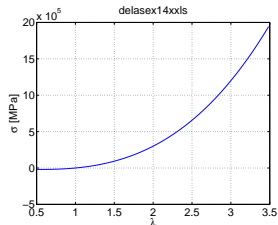
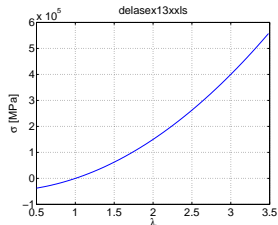
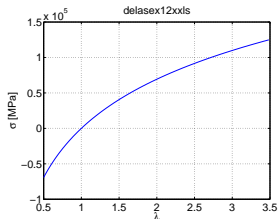
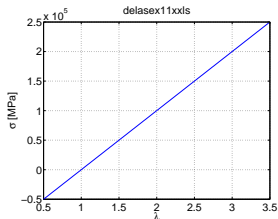


initial length	$l_0$	100	mm
initial cross-sectional area	$A_0$	10	mm <sup>2</sup>

# Elastic models : stress

elastic constant	$C$	100000	MPa
Poisson's ratio	$\nu$	0.3	-

$\sigma \sim \varepsilon_I$ model	$\sigma \sim \varepsilon_{In}$ model
$\sigma \sim \varepsilon_{gI}$ model	$P \sim \varepsilon_{GI}$ model

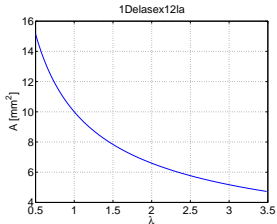
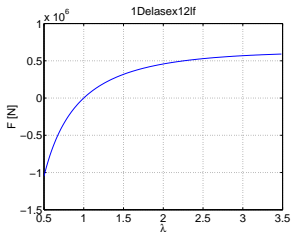
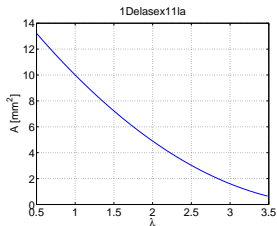
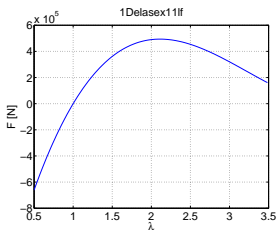


# Elastic models : force and area

elastic constant	$C$	100000	MPa
Poisson's ratio	$\nu$	0.3	-

$$\sigma \sim \varepsilon_I \text{ model}$$

$$\sigma \sim \varepsilon_{In} \text{ model}$$

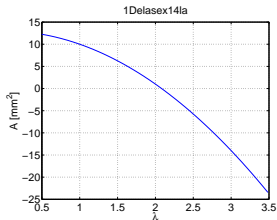
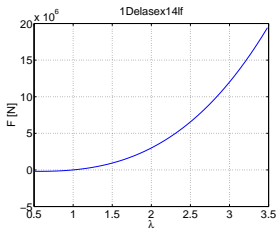
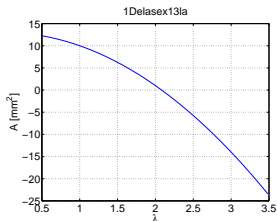
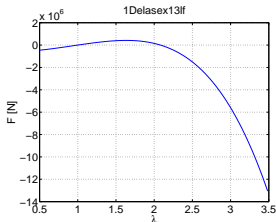




# Elastic models : force and area

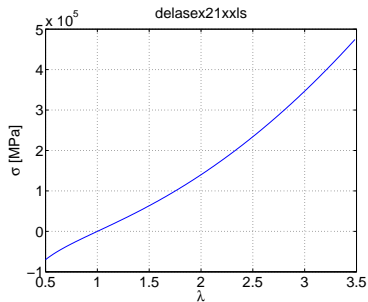
$$\sigma \sim \varepsilon_{gl} \text{ model}$$

$$P \sim \varepsilon_{gl} \text{ model}$$

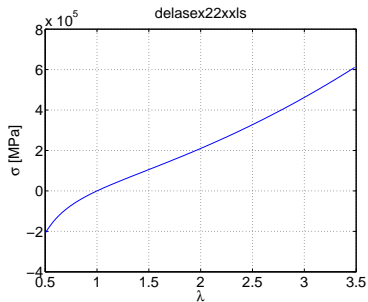


# Elastomeric models : stress

elastic constant	$C_{01}$	20000	MPa
elastic constant	$C_{10}$	20000	MPa



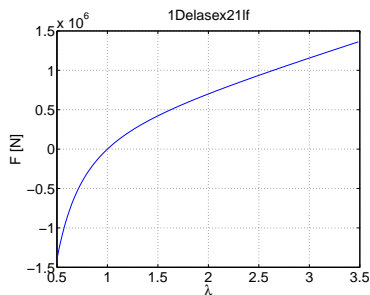
Neo-Hookean



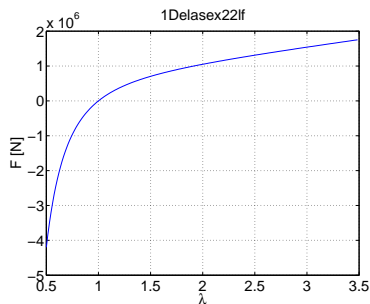
Mooney-Rivlin

# Elastomeric models : force and area

elastic constant	$C_{01}$	20000	MPa
elastic constant	$C_{10}$	20000	MPa



Neo-Hookean

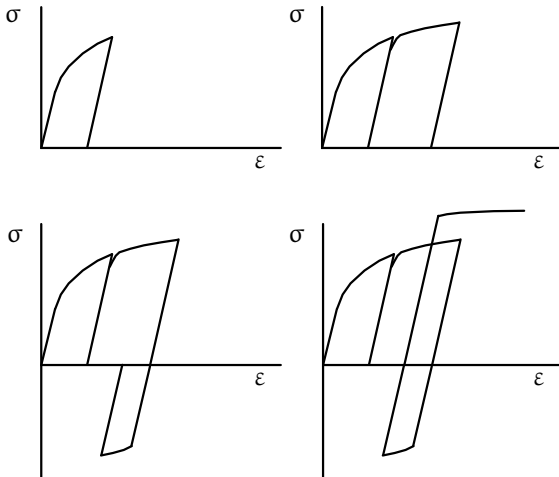


Mooney-Rivlin

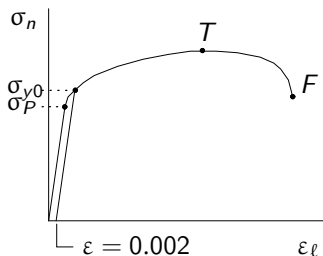
# ELASTOPLASTIC

[back to index](#)

# Elastoplastic material behavior



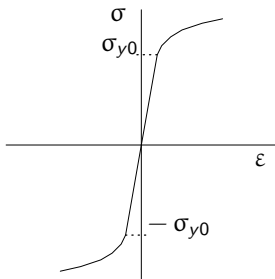
# Tensile test



$\sigma_P$	proportional limit
$\sigma_{y0}$	initial yield stress
$\varepsilon_{y0}$	strain at $\sigma_{y0}$ : $\varepsilon_{y0} = \sigma_{y0}/E$
$\varepsilon_{0.2}$	0.2-strain : $\varepsilon_p = 0.2\% = 0.002$
$\sigma_T$	tensile strength
$\sigma_F$	fracture strength
$\varepsilon_F$	fracture strain ( $\approx 5\% = 0.05$ (metals))

NB. : forming  $\rightarrow$  pressure  $\rightarrow$  larger strains

# Compression test



yield in tensile test

yield in compression test

yield general

elastic region

$$\sigma = \sigma_{y0}$$

$$\sigma = -\sigma_{y0}$$

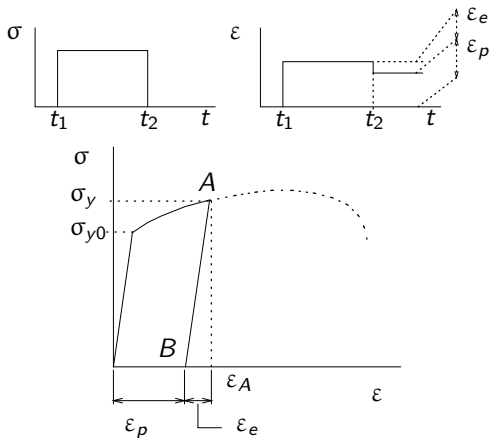
$$\sigma^2 = \sigma_{y0}^2$$

$$-\sigma_{y0} < \sigma < \sigma_{y0}$$

general yield criterion

$$f = \sigma^2 - \sigma_{y0}^2 = 0$$

# Interrupted tensile test



total strain  
plastic strain  
elastic strain  
assumptions

$$\epsilon = \epsilon_A$$

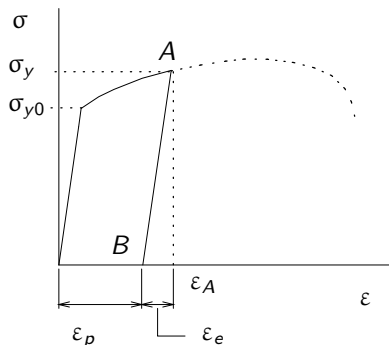
$$\epsilon_p$$

$$\epsilon_e \quad (\text{springback})$$

$$\text{elastic parameters constant} \rightarrow \Delta\sigma = E\Delta\epsilon = E\Delta\epsilon_e$$



# Resumed tensile test



linear behavior  $B \rightarrow A$

**current** yield stress

hardening

hardening model

history parameter

$$\Delta\sigma = E\Delta\varepsilon = E\Delta\varepsilon_e$$

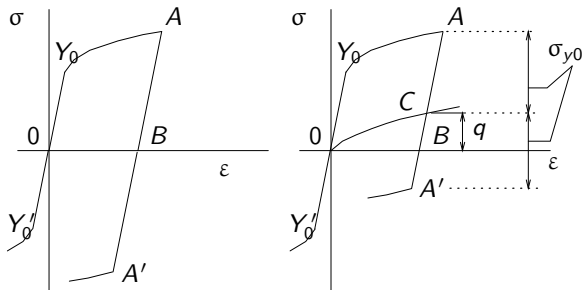
$$\sigma_y = \sigma_A$$

$$\sigma_y \text{ increases} \rightarrow \sigma_y > \sigma_{y0}$$

$$\sigma_y \sim \varepsilon_p$$

$$\varepsilon_p$$

# Hardening



isotropic hardening : elastic area larger & symmetric w.r.t.  $\sigma = 0$

$$\left. \begin{array}{l} \text{tensile} : \sigma = \sigma_y \\ \text{compression} : \sigma = -\sigma_y \end{array} \right\} \rightarrow f = \sigma^2 - \sigma_y^2 = 0$$

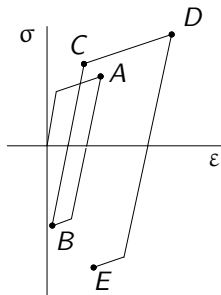
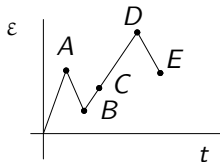
kinematic hardening : elastic area constant & symmetric w.r.t.  $\sigma = q$

$$\left. \begin{array}{l} \text{tensile} : \sigma = q + \sigma_{y0} \\ \text{compression} : \sigma = q - \sigma_{y0} \end{array} \right\} \rightarrow f = (\sigma - q)^2 - \sigma_{y0}^2 = 0$$

combined isotropic/kinematic hardening

$$\left. \begin{array}{l} \text{tensile} : \sigma = q + \sigma_y \\ \text{compression} : \sigma = q - \sigma_y \end{array} \right\} \rightarrow f = (\sigma - q)^2 - \sigma_y^2 = 0$$

# Effective plastic strain



$$\sigma_{yC} > \sigma_{yA} \quad ; \quad \varepsilon_{pC} < \varepsilon_{pA} \quad \rightarrow$$

effective plastic strain (rate)

$$\bar{\varepsilon}_p = \sum_{\varepsilon} |\Delta \varepsilon_p| = \sum_{\tau=0}^{\tau=t} \frac{|\Delta \varepsilon_p|}{\Delta t} \Delta t = \int_{\tau=0}^t |\dot{\varepsilon}_p| d\tau = \int_{\tau=0}^t \dot{\bar{\varepsilon}}_p d\tau$$

# Linear and power law hardening laws

linear hardening

$$\sigma_y = \sigma_{y0} + H\bar{\varepsilon}_p$$

Ludwik (1909)

$$\sigma_y = \sigma_{y0} + \sigma_{y0} \left( \frac{\bar{\varepsilon}_p}{\varepsilon_{y0}} \right)^n \quad (0 \leq n \leq 1) \quad \rightarrow$$

$$H = n \frac{\sigma_{y0}}{\varepsilon_{y0}} \left( \frac{\bar{\varepsilon}_p}{\varepsilon_{y0}} \right)^{n-1} = nE \left( \frac{\bar{\varepsilon}_p}{\varepsilon_{y0}} \right)^{n-1}$$

mod. Ludwik

$$\sigma_y = \sigma_{y0} (1 + m\bar{\varepsilon}_p^n) \quad \rightarrow \quad H = \sigma_{y0} mn\bar{\varepsilon}_p^{n-1}$$

Swift (1952)

$$\sigma_y = C(m + \bar{\varepsilon}_p)^n \quad \text{with} \quad C = \frac{\sigma_{y0}}{m^n}$$

$$H = Cn(m + \bar{\varepsilon}_p)^{n-1}$$

Ramberg-Osgood (1943)

$$\bar{\varepsilon}_p = \frac{\sigma_y}{E} \left[ 1 + \alpha \left( \frac{\sigma_y}{\sigma_{y0}} \right)^{m-1} \right]$$

$$(m \geq 0; \alpha = \frac{3}{7})$$

# Asymptotically perfect hardening laws

ideal plastic

$$\sigma_y = \sigma_{y0}$$

Prager (1938)

$$\sigma_y = \sigma_{y0} \tanh \left( \frac{E \bar{\epsilon}_p}{\sigma_{y0}} \right)$$

$$H = \frac{\sigma_{y0}}{\epsilon_{y0}} \left[ \operatorname{sech} \left( \frac{\bar{\epsilon}_p}{\epsilon_{y0}} \right) \right]^2 = E \left[ \operatorname{sech} \left( \frac{\bar{\epsilon}_p}{\epsilon_{y0}} \right) \right]^2$$

Betten I (1975)

$$\sigma_y = \sigma_{y0} \left[ \tanh \left( \frac{E \bar{\epsilon}_p}{\sigma_{y0}} \right)^m \right]^{1/m} \quad (m > 1)$$

$$H = E \left( \frac{\bar{\epsilon}_p}{\epsilon_{y0}} \right)^{m-1} \left[ \tanh \left( \frac{\bar{\epsilon}_p}{\epsilon_{y0}} \right)^m \right]^{\frac{1}{m}-1} \left[ \operatorname{sech} \left( \frac{\bar{\epsilon}_p}{\epsilon_{y0}} \right)^m \right]^2$$

Voce (1949)

$$\sigma_y = C (1 - n e^{-m \bar{\epsilon}_p}) \quad \text{with} \quad C = \frac{\sigma_{y0}}{1-n} \quad (m > 1)$$

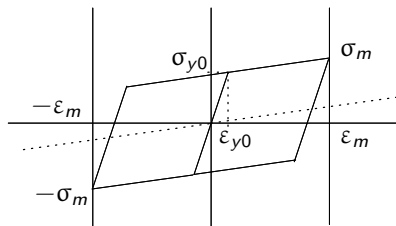
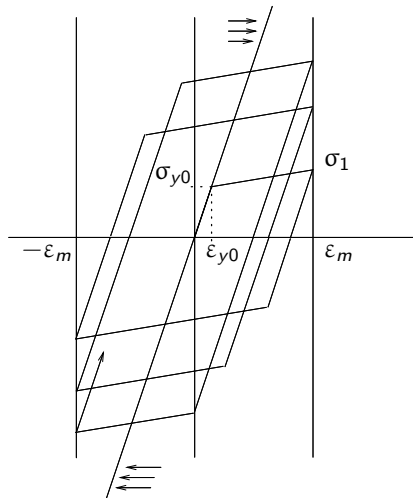
$$H = C n m e^{-m \bar{\epsilon}_p}$$

Betten II (1975)

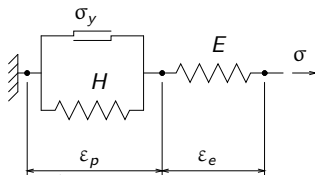
$$\sigma_y = \sigma_{y0} + (E \bar{\epsilon}_p) \left[ 1 + \left( \frac{\bar{\epsilon}_p}{\epsilon_{y0}} \right)^m \right]^{-1/m}$$

$$H = E \left[ 1 + \left( \frac{\bar{\epsilon}_p}{\epsilon_{y0}} \right)^m \right]^{-\frac{1}{m}} \left[ 1 - \left( \frac{\bar{\epsilon}_p}{\epsilon_{y0}} \right)^m \left\{ 1 + \left( \frac{\bar{\epsilon}_p}{\epsilon_{y0}} \right)^m \right\}^{-1} \right]$$

# Cyclic load



# Elastoplastic model



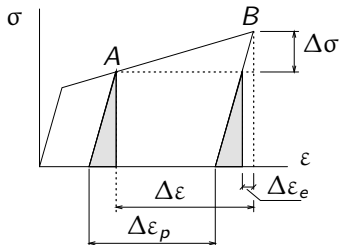
- $f = (\sigma - q)^2 - \sigma_y^2$  with  $f < 0 \mid f = 0 \wedge \dot{f} < 0 \rightarrow \text{el.}$   
 $f = 0 \wedge \dot{f} = 0 \rightarrow \text{elpl.}$
- $\sigma_y = \sigma_y(\sigma_{y0}, \bar{\epsilon}_p) \quad ; \quad q = q(\epsilon_p)$
- $\dot{\epsilon} = \dot{\epsilon}_e + \dot{\epsilon}_p$
- $\sigma = E\epsilon_e \rightarrow \dot{\epsilon}_e = \frac{1}{E} \dot{\sigma}$
- $\dot{\epsilon}_p = \dot{\lambda} \frac{\partial f}{\partial \sigma} = 2\dot{\lambda}(\sigma - q) \quad ; \quad \dot{\bar{\epsilon}}_p = |\dot{\epsilon}_p| = 2\dot{\lambda}|\sigma - q|$
- $\bar{\epsilon}_p = \int_{\tau=0}^t \dot{\bar{\epsilon}}_p d\tau = \sum_t |\Delta\epsilon_p|$

# Constitutive equations

$$\left. \begin{aligned} \dot{\sigma} &= E\dot{\epsilon}_e = E(\dot{\epsilon} - \dot{\epsilon}_p) = E\{\dot{\epsilon} - 2\dot{\lambda}(\sigma - q)\} \\ f &= 0 \end{aligned} \right\} \rightarrow$$
$$\left. \begin{aligned} \dot{\sigma} + 2E(\sigma - q)\dot{\lambda} - E\dot{\epsilon} &= 0 \\ f &= 0 \end{aligned} \right\}$$



# Isotropic hardening "monotonic" tensile test $A \rightarrow B$



$$\begin{aligned}\Delta\sigma &= E\Delta\epsilon_e = E(\Delta\epsilon - \Delta\epsilon_p) \\ &= E\left(\Delta\epsilon - \frac{\Delta\sigma_y}{H}\right) = E\left(\Delta\epsilon - \frac{\Delta\sigma}{H}\right) \rightarrow \\ \Delta\sigma &= \frac{EH}{E+H} \Delta\epsilon = S\Delta\epsilon \quad ; \quad \Delta\epsilon_p = \frac{\Delta\sigma}{H} = \frac{E}{E+H} \Delta\epsilon\end{aligned}$$

## Kinematic hardening

$$\Delta\sigma = \frac{EK}{E+K} \Delta\varepsilon \quad ; \quad \Delta\varepsilon_p = \frac{1}{K} \Delta\sigma = \frac{E}{E+K} \Delta\varepsilon$$

$$\Delta\sigma = E\Delta\varepsilon_e = E(\Delta\varepsilon - \Delta\varepsilon_p)$$

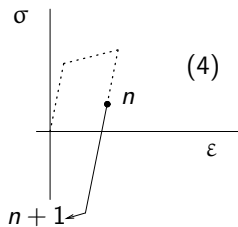
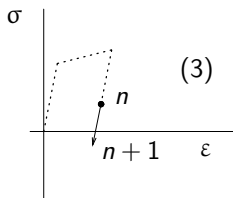
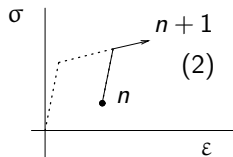
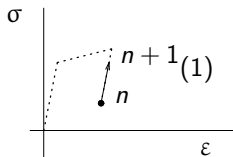
$$= E \left( \Delta\varepsilon - \frac{\Delta q}{K} \right) = E \left( \Delta\varepsilon - \frac{\Delta\sigma}{K} \right) \quad \rightarrow$$

$$\Delta\sigma = \frac{EK}{E+K} \Delta\varepsilon = S\Delta\varepsilon \quad ; \quad \Delta\varepsilon_p = \frac{\Delta\sigma}{K} = \frac{E}{E+K} \Delta\varepsilon$$

$$\lim_{H \rightarrow \infty} \frac{EH}{E+H} = \lim_{H \rightarrow \infty} \frac{E}{\frac{E}{H} + 1} = E$$

# Stress update

WHERE ARE WE ??



# Elastic stress predictor

$$\sigma_e = \sigma_n + E(\varepsilon - \varepsilon_n)$$

- $f = (\sigma_e - q_n)^2 - \sigma_{y_n}^2 \leq 0 \quad \rightarrow \quad \text{elastic increment}$
- $f = (\sigma_e - q_n)^2 - \sigma_{y_n}^2 > 0 \quad \rightarrow \quad \text{elastoplastic increment}$

# Elastic increment

- elastic solution is end-increment solution
- continue loading history

$$\sigma(t_{n+1}) = \sigma_e \quad ; \quad \bar{\varepsilon}_p(t_{n+1}) = \bar{\varepsilon}_p(t_n) = \bar{\varepsilon}_{p_n}$$

$$\sigma_y(t_{n+1}) = \sigma_y(t_n) = \sigma_{y_n} \quad ; \quad q(t_{n+1}) = q(t_n) = q_n$$

# Implicit solution procedure

$$\left. \begin{aligned} \sigma - \sigma_n + 2E(\sigma - q)(\lambda - \lambda_n) &= E(\varepsilon - \varepsilon_n) \\ f - f_n &= f = 0 \end{aligned} \right\}$$

$$\left. \begin{aligned} \sigma^* + \delta\sigma - \sigma_n + 2E(\sigma^* + \delta\sigma - q^* - \delta q)(\lambda^* + \delta\lambda - \lambda_n) &= E(\varepsilon - \varepsilon_n) \\ f^* + \delta f = 0 \quad \rightarrow \quad f^* + \frac{\partial f}{\partial \sigma} \delta\sigma + \frac{\partial f}{\partial \lambda} \delta\lambda &= 0 \end{aligned} \right\}$$

$$\frac{\partial f}{\partial \sigma} = 2(\sigma - q)$$

$$\begin{aligned} \frac{\partial f}{\partial \lambda} &= \frac{\partial f}{\partial q} \frac{\partial q}{\partial \varepsilon_p} \frac{\partial \varepsilon_p}{\partial \lambda} + \frac{\partial f}{\partial \sigma_y} \frac{\partial \sigma_y}{\partial \bar{\varepsilon}_p} \frac{\partial \bar{\varepsilon}_p}{\partial \lambda} \\ &= [-2(\sigma - q)][K][2(\sigma - q)] + [-2\sigma_y][H][2|\sigma - q|] \\ &= -4K(\sigma - q)^2 - 4H\sigma_y|\sigma - q| \end{aligned}$$

$$\left. \begin{aligned} \sigma^* + \delta\sigma - \sigma_n + 2E(\sigma^* + \delta\sigma - q^* - \delta q)(\lambda^* + \delta\lambda - \lambda_n) &= E(\varepsilon - \varepsilon_n) \\ f^* + 2(\sigma^* - q^*)\delta\sigma - [4K^*(\sigma^* - q^*)^2 + 4H^*\sigma_y^*|\sigma^* - q^*|]\delta\lambda &= 0 \end{aligned} \right\}$$

# Implicit solution procedure

$$\left. \begin{aligned}\sigma^* + \delta\sigma - \sigma_n + 2E(\sigma^* + \delta\sigma - q^* - \delta q)(\lambda^* + \delta\lambda - \lambda_n) &= E(\varepsilon - \varepsilon_n) \\ f^* + 2(\sigma^* - q^*)\delta\sigma - [4K^*(\sigma^* - q^*)^2 + 4H^*\sigma_y^*|\sigma^* - q^*|]\delta\lambda &= 0\end{aligned}\right\}$$

$$\sigma^* = \sigma^* + \delta\sigma$$

$$\lambda^* = \lambda^* + \delta\lambda$$

$$\Delta\varepsilon_p = 2(\lambda^* - \lambda_n)(\sigma^* - q_n) \rightarrow \varepsilon_p \rightarrow q^*, K^*$$

$$\Delta\bar{\varepsilon}_p = |\Delta\varepsilon_p| \rightarrow \bar{\varepsilon}_p \rightarrow \sigma_y^*, H^*$$



## Stiffness : implicit

$$\begin{cases} \sigma - \sigma_n + 2E(\sigma - q)(\lambda - \lambda_n) - E(\varepsilon - \varepsilon_n) = 0 \\ f = 0 \end{cases}$$

$$\begin{cases} \delta\sigma + 2E\delta\sigma(\lambda - \lambda_n) + 2E(\sigma - q)\delta\lambda - E\delta\varepsilon = 0 \\ (\sigma - q)\delta\sigma - 2K(\sigma - q)^2\delta\lambda - 2H\sigma_y|\sigma - q|\delta\lambda = 0 \end{cases}$$

$$\left[ 1 + 2E(\lambda - \lambda_n) + \frac{2E(\sigma - q)^2}{2K(\sigma - q)^2 + 2H\sigma_y|\sigma - q|} \right] \delta\sigma = E\delta\varepsilon$$

$$C_\varepsilon = \frac{E\{2K(\sigma - q)^2 + 2H\sigma_y|\sigma - q|\}}{\{1 + 2E(\lambda - \lambda_n)\}[2K(\sigma - q)^2 + 2H\sigma_y|\sigma - q|] + 2E(\sigma - q)^2}$$

$$\text{yield at } \tau = t = t_{n+1} \quad \rightarrow \quad (\sigma - q)^2 = \sigma_y^2 \text{ and } |\sigma - q| = \sigma_y \quad \rightarrow$$

$$C_\varepsilon = \frac{E(K + H)}{E + K + H + 2E(K + H)(\lambda - \lambda_n)}$$

# Explicit solution procedure

$$\left. \begin{aligned} \Delta\sigma + 2E(\sigma_n - q_n)\Delta\lambda &= E\Delta\varepsilon \\ \Delta f = 0 &\rightarrow \left. \frac{\partial f}{\partial \sigma} \right|_n \Delta\sigma + \left. \frac{\partial f}{\partial \lambda} \right|_n \Delta\lambda = 0 \end{aligned} \right\}$$

$$\Delta\sigma + 2E(\sigma_n - q_n)\Delta\lambda = E\Delta\varepsilon$$

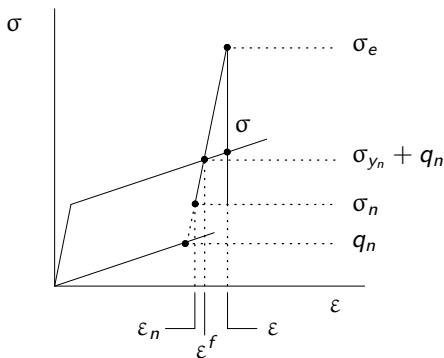
$$2(\sigma_n - q_n)\Delta\sigma - 4K_n(\sigma_n - q_n)^2\Delta\lambda - 4H_n\sigma_{yn}|\sigma_n - q_n|\Delta\lambda = 0 \rightarrow$$

$$\Delta\lambda = \frac{(\sigma_n - q_n)}{2K_n(\sigma_n - q_n)^2 + 2H_n\sigma_{yn}|\sigma_n - q_n|} \Delta\sigma = \frac{1}{2K_n(\sigma_n - q_n) + 2H_n(\sigma_n - q_n)} \Delta\sigma$$

$$\Delta\sigma = \frac{E[K_n(\sigma_n - q_n)^2 + H_n\sigma_{yn}|\sigma_n - q_n|]}{K_n(\sigma_n - q_n)^2 + H_n\sigma_{yn}|\sigma_n - q_n| + E(\sigma_n - q_n)^2} \Delta\varepsilon$$

$$\Delta\varepsilon_p = 2(\sigma_n - q_n)\Delta\lambda = \frac{(\sigma_n - q_n)^2}{K_n(\sigma_n - q_n)^2 + H_n\sigma_{yn}|\sigma_n - q_n|}$$

# Increment splitting



$$\sigma_e = \sigma_n + E(\epsilon - \epsilon_n) \quad \rightarrow \quad \Delta\sigma_e = \sigma_e - \sigma_n = E(\epsilon - \epsilon_n)$$

$$\beta = \frac{|\text{sign}(\epsilon - \epsilon_n)\sigma_{y_n} - (\sigma_n - q_n)|}{|\sigma_e - \sigma_n|}$$

$$\epsilon^f = \epsilon_n + \beta(\epsilon - \epsilon_n) \quad \rightarrow \quad \Delta\epsilon^f = \epsilon - \epsilon^f = (1 - \beta)(\epsilon - \epsilon_n)$$

## Explicit stress update

$$\left\{ \begin{array}{l} \Delta\sigma^f + 2E(\sigma_n - q)\Delta\lambda = E\Delta\varepsilon^f \\ 2(\sigma_n - q)\Delta\sigma^f - 4K_n(\sigma_n - q_n)^2\Delta\lambda - 4H_n\sigma_{yn}|\sigma_n - q_n|\Delta\lambda = 0 \end{array} \right.$$

$$\Delta\sigma = \beta\Delta\sigma_e + \Delta\sigma^f \quad \rightarrow \quad \sigma = \sigma_n + \Delta\sigma$$

$$\lambda = \lambda_n + \Delta\lambda$$

$$\Delta\varepsilon_p = 2(\lambda - \lambda_n)(\sigma - q_n) \quad \rightarrow \quad \varepsilon_p \quad \rightarrow \quad q, K$$

$$\Delta\bar{\varepsilon}_p = |\Delta\varepsilon_p| \quad \rightarrow \quad \bar{\varepsilon}_p \quad \rightarrow \quad \sigma_y, H$$

# Implementation

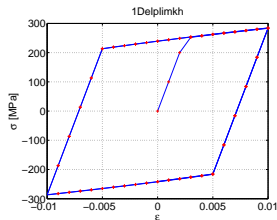
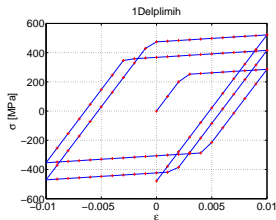
`tr2delp1.m`

# Cyclic loading

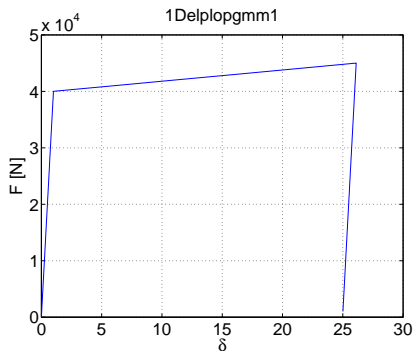
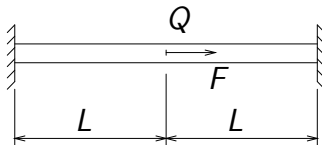


initial length	$l_0$	100	mm
initial cross-sectional area	$A_0$	10	mm <sup>2</sup>

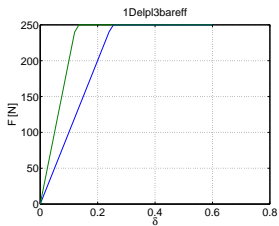
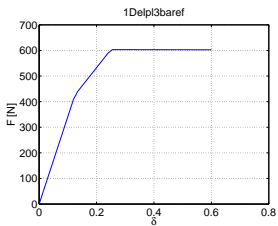
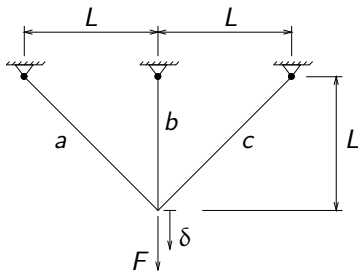
Young's modulus	$E$	100000	MPa
Poisson's ratio	$\nu$	0.3	-
initial yield stress	$\sigma_{y0}$	250	MPa
isotropic hardening coefficient	$H$	5000	MPa
kinematic hardening coefficient	$K$	5000	MPa



# Clamped truss

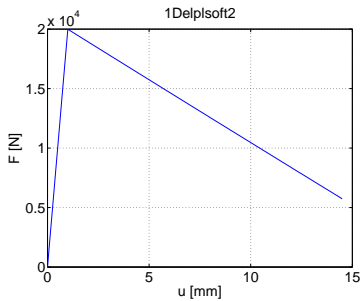
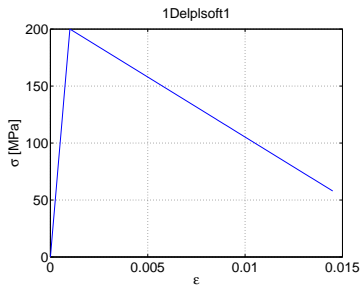


# Truss structure





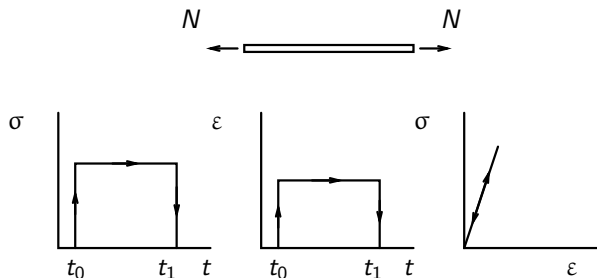
# Softening material



## LINEAR VISCOELASTIC

[back to index](#)

# Linear elastic material behavior



$$\epsilon = \frac{1}{E} \sigma \quad \rightarrow \quad \sigma = E \epsilon \quad \rightarrow$$

$$N = \sigma A = EA \epsilon = \frac{EA}{l} \Delta l = k \Delta l$$

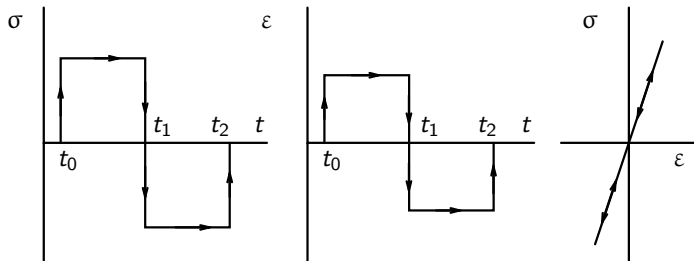
constant Young's modulus

$E$  : Hooke's law

linear spring : spring stiffness

$$k = \frac{EA}{l}$$

# Load cycle

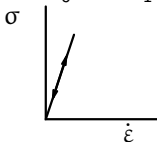
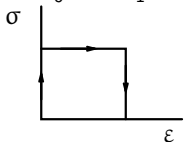
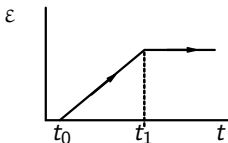
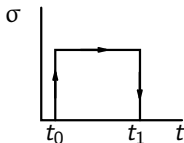


no dissipation : no area under  $(\sigma, \epsilon)$ -curve

$$\begin{aligned} U_d &= \int_{t_0}^{t_1} \sigma d\epsilon + \int_{t_1}^{t_2} \sigma d\epsilon = \int_{t_0}^{t_1} E\epsilon d\epsilon + \int_{t_1}^{t_2} E\epsilon d\epsilon \\ &= \frac{1}{2}E[\epsilon_1^2 - \epsilon_0^2 + \epsilon_2^2 - \epsilon_1^2] = 0 \end{aligned}$$

NO ENERGY DISSIPATION

# Linear viscous material behavior



$$\dot{\varepsilon} = \frac{1}{\eta} \sigma \quad \rightarrow \quad \sigma = \eta \dot{\varepsilon} \quad \rightarrow$$

$$N = \sigma A = \eta A \dot{\varepsilon} = \frac{\eta A}{l} \dot{\Delta l} = b \dot{\Delta l}$$

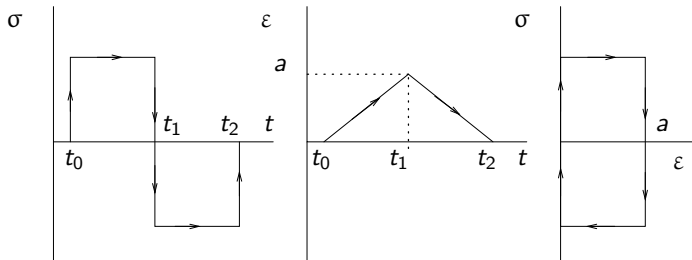
constant viscosity

$\eta$  : Newtonian fluid

linear dashpot : damping constant

$$b = \frac{\eta A}{l}$$

# Load cycle

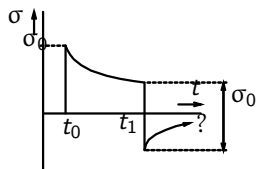
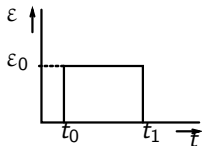
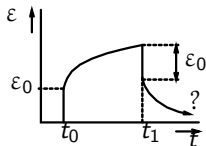
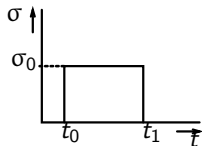


dissipated energy  $\sim$  area

$$\begin{aligned}
 U_d &= \int_{t_0}^{t_1} \sigma d\epsilon + \int_{t_1}^{t_2} \sigma d\epsilon = \int_{t_0}^{t_1} \eta \dot{\epsilon} d\epsilon + \int_{t_1}^{t_2} \eta \dot{\epsilon} d\epsilon = \int_{t_0}^{t_1} \eta c d\epsilon - \int_{t_1}^{t_2} \eta c d\epsilon \\
 &= \eta c [\epsilon_1 - \epsilon_0 - \epsilon_2 + \epsilon_1] = 2\eta c a
 \end{aligned}$$

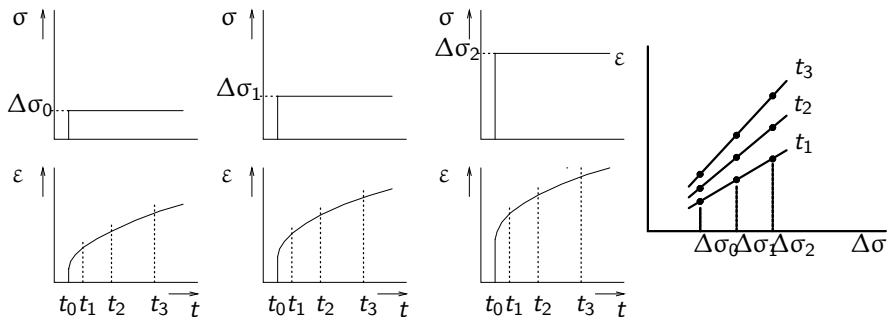
TOTAL ENERGY DISSIPATION

# Viscoelastic material behavior



- small deformations !!
- description of experimental observations
- modeling the material behavior (a.o. with spring-dashpot models)

# Proportionality



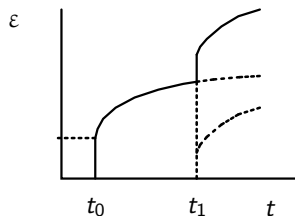
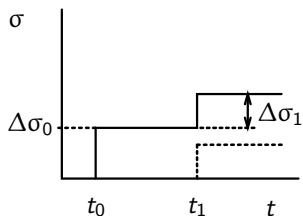
linear **isochrones**  $\rightarrow$  proportionality

$$\epsilon(t) = \Delta\sigma D(t - t_0) \quad \text{for} \quad \forall \quad t \geq t_0$$

$D(t - t_0)$  is no function of the stresses



# Superposition



separate excitations

$$\Delta\sigma = \Delta\sigma_0 \quad \rightarrow \quad \epsilon(t) = \Delta\sigma_0 D(t - t_0) \quad \text{for} \quad t > t_0$$

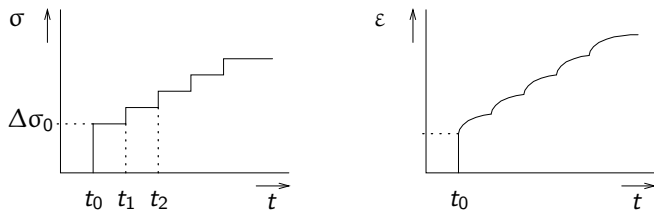
$$\Delta\sigma = \Delta\sigma_1 \quad \rightarrow \quad \epsilon(t) = \Delta\sigma_1 D(t - t_1) \quad \text{for} \quad t > t_1$$

subsequent excitations

$$\Delta\sigma = \Delta\sigma_0 \quad \rightarrow \quad \epsilon(t) = \Delta\sigma_0 D(t - t_0) \quad \text{for} \quad t_0 < t < t_1$$

$$\Delta\sigma = \Delta\sigma_0 + \Delta\sigma_1 \quad \rightarrow \quad \epsilon(t) = \Delta\sigma_0 D(t - t_0) + \Delta\sigma_1 D(t - t_1) \quad \text{for} \quad t > t_1$$

# Boltzmann integral : strain respons



$$\varepsilon(t) = \Delta\sigma_0 D(t - t_0) + \Delta\sigma_1 D(t - t_1) + \Delta\sigma_2 D(t - t_2) + ..$$

$$= \sum_{i=1}^n \Delta\sigma_i D(t - t_i) \quad \rightarrow \quad \text{limit } n \rightarrow \infty \quad (t \rightarrow \tau)$$

$$= \int_{\tau=t_0^-}^t D(t - \tau) d\sigma(\tau) = \int_{\tau=t_0^-}^t D(t - \tau) \frac{d\sigma(\tau)}{d\tau} d\tau$$

$$\varepsilon(t) = \int_{\tau=t_0^-}^t D(t - \tau) \dot{\sigma}(\tau) d\tau$$

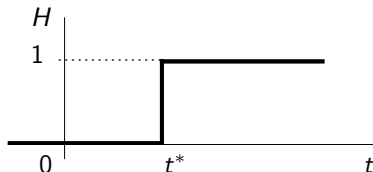
## Boltzmann integral : stress respons

$$\sigma(t) = \int_{\tau=t_0^-}^t E(t-\tau) \dot{\epsilon}(\tau) d\tau$$

# Step excitations

Heaviside function

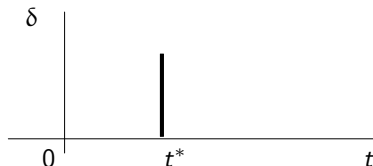
$$H(t, t^*) = \begin{cases} t < t^* & : H(t, t^*) = 0 \\ t > t^* & : H(t, t^*) = 1 \end{cases}$$



Dirac function

$$\delta(t, t^*) = \frac{d}{dt} \{H(t, t^*)\}$$

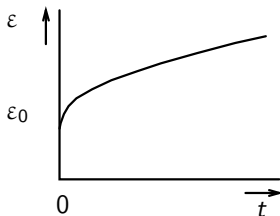
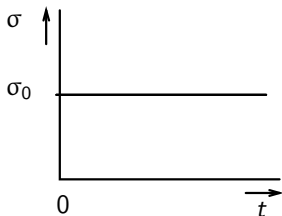
$$\int_{\tau=0}^{t > t^*} \delta(\tau, t^*) d\tau = 1 \quad ; \quad \int_{\tau=0}^{t > t^*} f(\tau) \delta(\tau, t^*) d\tau = f(t^*)$$



# Creep (Retardation) $\rightarrow$ creep function

$$\sigma(t) = \sigma_0 H(t, 0) \quad \rightarrow \quad \dot{\sigma}(t) = \sigma_0 \delta(t, 0)$$

$$\varepsilon(t) = \int_{\tau=0-}^t D(t-\tau) \dot{\sigma}(\tau) d\tau = \int_{\tau=0-}^t D(t-\tau) \sigma_0 \delta(\tau, 0) d\tau = \sigma_0 D(t)$$

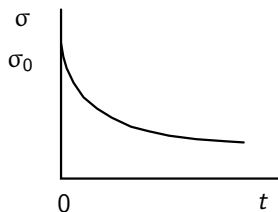
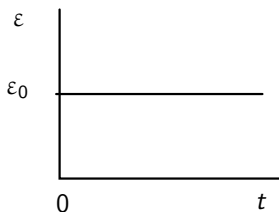


- $\dot{D}(t) \geq 0 \quad \forall \quad t \geq 0$
- $\ddot{D}(t) < 0 \quad \forall \quad t \geq 0$

# Relaxation → relaxation function

$$\varepsilon(t) = \varepsilon_0 H(t, 0) \quad \rightarrow \quad \dot{\varepsilon}(t) = \varepsilon_0 \delta(t, 0)$$

$$\sigma(t) = \int_{\tau=0^-}^t E(t-\tau) \dot{\varepsilon}(\tau) d\tau = \int_{\tau=0^-}^t E(t-\tau) \varepsilon_0 \delta(\tau, 0) d\tau = \varepsilon_0 E(t)$$

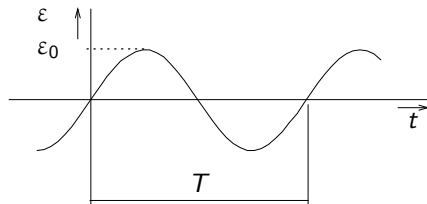


- $\dot{E}(t) \leq 0 \quad \forall \quad t \geq 0$
- $\ddot{E}(t) > 0 \quad \forall \quad t \geq 0$
- $\int_{t=0}^{\infty} \dot{E}(t) dt \geq 0 \quad \rightarrow \quad \lim_{t \rightarrow \infty} \dot{E}(t) = 0$

# Harmonic strain excitation

harmonic strain ( $\omega = \text{angular frequency [rad s}^{-1}\text{]}$ )

$$\varepsilon(t) = \varepsilon_0 \sin(\omega t) \rightarrow \dot{\varepsilon}(t) = \varepsilon_0 \omega \cos(\omega t)$$



amplitude

$\varepsilon_0$

angular frequency

$\omega \text{ [rad s}^{-1}\text{]}$

period and frequency

$$T = \frac{2\pi}{\omega} \text{ [s}^{-1}\text{]} \quad ; \quad f = \frac{1}{T}$$

# Stress response

$$\begin{aligned}\sigma(t) &= \int_{\tau=-\infty}^t E(t-\tau) \varepsilon_0 \omega \cos(\omega \tau) d\tau \\&= \varepsilon_0 \omega \int_{\xi=-\infty}^t E(t-\tau) \cos(\omega \tau) d\tau \\&\quad t - \tau = s \quad \rightarrow \quad \tau = t - s \quad \rightarrow \quad d\tau = -ds \\&= \varepsilon_0 \omega \int_{s=0}^{\infty} E(s) \cos\{\omega(t-s)\} ds \\&\quad \cos(\omega t - \omega s) = \cos(\omega t) \cos(\omega s) + \sin(\omega t) \sin(\omega s) \\&= \varepsilon_0 \left[ \omega \int_{s=0}^{\infty} E(s) \sin(\omega s) ds \right] \sin(\omega t) + \varepsilon_0 \left[ \omega \int_{s=0}^{\infty} E(s) \cos(\omega s) ds \right] \cos(\omega t) \\&= \varepsilon_0 E' \sin(\omega t) + \varepsilon_0 E'' \cos(\omega t)\end{aligned}$$

$$E'(\omega) = \omega \int_{s=0}^{\infty} E(s) \sin(\omega s) ds \quad : \text{ storage modulus}$$

$$E''(\omega) = \omega \int_{s=0}^{\infty} E(s) \cos(\omega s) ds \quad : \text{ loss modulus}$$



# Energy dissipation

one period

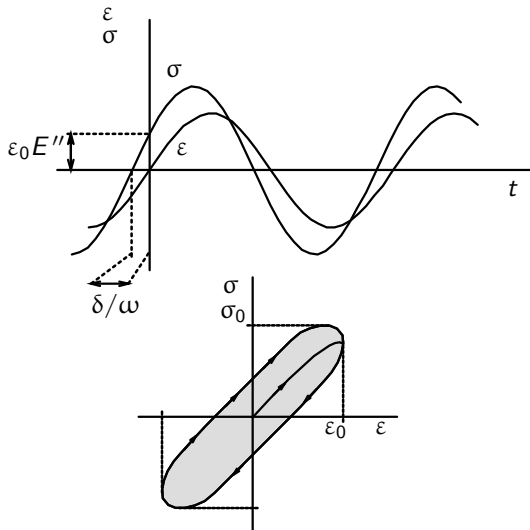
$$0 \leq t \leq \frac{2\pi}{\omega} = T = \frac{1}{f}$$

dissipated energy per unit of volume

$$\begin{aligned} U_d &= \int_{\varepsilon(0)}^{\varepsilon(T)} \sigma d\varepsilon = \int_{t=0}^T \sigma \dot{\varepsilon} dt \\ &= \int_{t=0}^T \{ \varepsilon_0 E' \sin(\omega t) + \varepsilon_0 E'' \cos(\omega t) \} \{ \varepsilon_0 \omega \cos(\omega t) \} dt \\ &= \int_{t=0}^T \varepsilon_0^2 \omega \{ E' \sin(\omega t) \cos(\omega t) + E'' \cos^2(\omega t) \} dt \\ &= \int_{t=0}^T \varepsilon_0^2 \omega \left\{ \frac{1}{2} E' \sin(2\omega t) + \frac{1}{2} E'' + \frac{1}{2} E'' \cos(2\omega t) \right\} dt \\ &= \frac{1}{2} \varepsilon_0^2 \omega \left[ -E' \frac{1}{2\omega} \cos(2\omega t) + E'' t + E'' \frac{1}{2\omega} \sin(2\omega t) \right]_0^{T=\frac{2\pi}{\omega}} \\ &= \frac{1}{2} \varepsilon_0^2 \omega \left[ -E' \frac{1}{2\omega} + E' \frac{1}{2\omega} + E'' \frac{2\pi}{\omega} \right] = \pi \varepsilon_0^2 E'' \\ &> 0 \quad \Rightarrow \quad E'' > 0 \quad \rightarrow \end{aligned}$$

# Phase difference

phase angle  $\delta$  (phase difference  $\frac{\delta}{\omega}$ )



hysteresis

## Relation between $E'$ , $E''$ and $\delta_\sigma$

stress response  $(\delta = \text{phase angle})$

$$\begin{aligned}\sigma(t) &= \sigma_0 \sin(\omega t + \delta) \\ &= \sigma_0 \cos(\delta) \sin(\omega t) + \sigma_0 \sin(\delta) \cos(\omega t) \\ \sigma(t) &= \varepsilon_0 E' \sin(\omega t) + \varepsilon_0 E'' \cos(\omega t)\end{aligned}$$

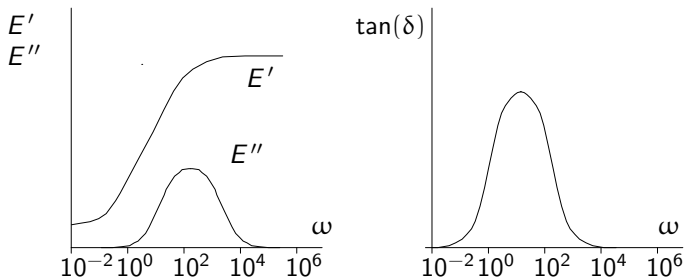
storage and loss modulus

$$\left. \begin{aligned}E' &= \frac{\sigma_0}{\varepsilon_0} \cos(\delta) \\ E'' &= \frac{\sigma_0}{\varepsilon_0} \sin(\delta)\end{aligned} \right\} \rightarrow \left\{ \begin{aligned}\frac{E''}{E'} &= \tan(\delta) \rightarrow \\ \delta &= \arctan\left(\frac{E''}{E'}\right)\end{aligned} \right.$$

amplitude

$$\sigma_0 = \varepsilon_0 \sqrt{(E')^2 + (E'')^2}$$

## Measured $E'$ , $E''$ and $\tan(\delta)$



- measurement of  $E'$  and  $E''$  can be done accurately
- $E'(\omega), E''(\omega) \rightarrow E(t)$  via fitting procedure
- range  $\omega \rightarrow$  temperature  $\rightarrow$  DMTA
- measurement of  $E(t)$  in relaxation test is difficult
- fit is inaccurate

# Harmonic stress excitation

$$\sigma(t) = \sigma_0 \sin(\omega t) \quad \rightarrow \quad \dot{\sigma}(t) = \sigma_0 \omega \cos(\omega t)$$

$$\begin{aligned}\varepsilon(t) &= \int_{\tau=-\infty}^t D(t-\tau) \dot{\sigma}(\tau) d\tau \\&= \int_{\tau=-\infty}^t D(t-\tau) \sigma_0 \omega \cos(\omega \tau) d\tau \\&= \sigma_0 \left[ \omega \int_{s=0}^{\infty} D(s) \sin(\omega s) ds \right] \sin(\omega t) + \sigma_0 \left[ \omega \int_{s=0}^{\infty} D(s) \cos(\omega s) ds \right] \cos(\omega t) \\&= \sigma_0 D' \sin(\omega t) - \sigma_0 D'' \cos(\omega t)\end{aligned}$$

$$D'(\omega) = \omega \int_{s=0}^{\infty} D(s) \sin(\omega s) ds \quad : \quad \text{storage compliance}$$

$$D''(\omega) = -\omega \int_{s=0}^{\infty} D(s) \cos(\omega s) ds \quad : \quad \text{loss compliance}$$

# Relation between $D'$ , $D''$ and $\delta_\varepsilon$

strain response

$$\begin{aligned}\varepsilon(t) &= \varepsilon_0 \sin(\omega t - \delta) \\ &= \varepsilon_0 \cos(\delta) \sin(\omega t) - \varepsilon_0 \sin(\delta) \cos(\omega t) \\ \varepsilon(t) &= \sigma_0 D' \sin(\omega t) - \sigma_0 D'' \cos(\omega t)\end{aligned}$$

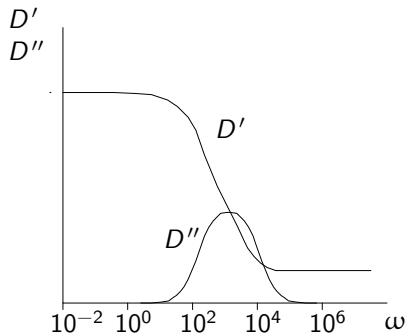
storage and loss compliance

$$\left. \begin{aligned} D' &= \frac{\varepsilon_0}{\sigma_0} \cos(\delta) \\ D'' &= \frac{\varepsilon_0}{\sigma_0} \sin(\delta) \end{aligned} \right\} \rightarrow \left\{ \begin{aligned} \frac{D''}{D'} &= \tan(\delta) \rightarrow \\ \delta &= \arctan\left(\frac{D''}{D'}\right) \end{aligned} \right.$$

amplitude

$$\varepsilon_0 = \sigma_0 \sqrt{(D')^2 + (D'')^2}$$

## Measured $D'$ and $D''$



## Relation between $(D', D'')$ and $(E', E'')$

$$\left. \begin{aligned} \sigma_0 &= \varepsilon_0 \sqrt{(E')^2 + (E'')^2} \\ \varepsilon_0 &= \sigma_0 \sqrt{(D')^2 + (D'')^2} \end{aligned} \right\} \rightarrow$$
$$[(E')^2 + (E'')^2][(D')^2 + (D'')^2] = 1 \quad (1)$$

$$\frac{D''}{D'} = \frac{E''}{E'} \rightarrow D'' = D' \frac{E''}{E'} \quad (2)$$

$$(1) \ \& \ (2) \rightarrow D' = \frac{E'}{(E')^2 + (E'')^2} \quad ; \quad D'' = \frac{E''}{(E')^2 + (E'')^2}$$

$$\text{idem} \quad E' = \frac{D'}{(D')^2 + (D'')^2} \quad ; \quad E'' = \frac{D''}{(D')^2 + (D'')^2}$$



# Complex variables

$$\varepsilon(t) = \varepsilon_0 \sin(\omega t) = \varepsilon_0 \cos(\omega t - \frac{\pi}{2}) = \operatorname{Re} \left[ \varepsilon_0 e^{-i\frac{\pi}{2}} e^{i\omega t} \right] = \operatorname{Re} [\varepsilon^* e^{i\omega t}]$$

$$\sigma(t) = \sigma_0 \sin(\omega t + \delta) = \sigma_0 \cos(\omega t - \frac{\pi}{2} + \delta) = \operatorname{Re} \left[ \sigma_0 e^{i(\delta - \frac{\pi}{2})} e^{i\omega t} \right] = \operatorname{Re} [\sigma^* e^{i\omega t}]$$

complex modulus and compliance

$$E^* = \frac{\sigma^*}{\varepsilon^*} = \frac{\sigma_0}{\varepsilon_0} e^{i\delta} = \frac{\sigma_0}{\varepsilon_0} \cos(\delta) + i \frac{\sigma_0}{\varepsilon_0} \sin(\delta) = E' + iE''$$

$$D^* = \frac{\varepsilon^*}{\sigma^*} = \frac{\varepsilon_0}{\sigma_0} e^{-i\delta} = \frac{\varepsilon_0}{\sigma_0} \cos(\delta) - i \frac{\varepsilon_0}{\sigma_0} \sin(\delta) = D' - iD''$$

dynamic modulus en compliance

$$E_d = |E^*| = \sqrt{(E')^2 + (E'')^2} = \frac{\sigma_0}{\varepsilon_0}$$

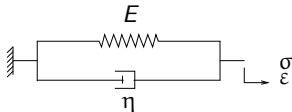
$$D_d = |D^*| = \sqrt{(D')^2 + (D'')^2} = \frac{\varepsilon_0}{\sigma_0}$$

# Viscoelastic models

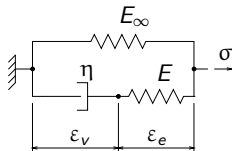
Maxwell



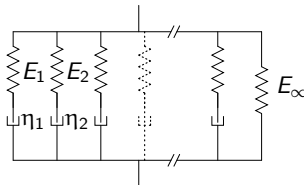
Kelvin-Voigt



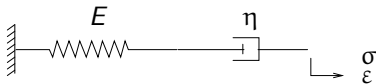
Standard Solid



Generalized Maxwell



# Maxwell model



$$\varepsilon = \varepsilon_E + \varepsilon_\eta \quad \rightarrow$$

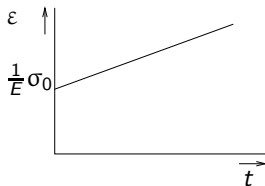
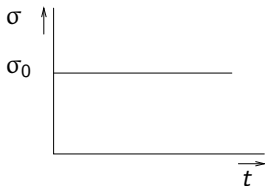
$$\dot{\varepsilon} = \dot{\varepsilon}_E + \dot{\varepsilon}_\eta = \frac{\dot{\sigma}}{E} + \frac{\sigma}{\eta}$$

## Maxwell : stress step excitation

$$\sigma(t) = \sigma_0 H(t, 0) \quad \rightarrow \quad \dot{\sigma}(t) = \sigma_0 \delta(t, 0)$$

$$\dot{\varepsilon}(t) = \frac{\sigma_0}{E} \delta(t, 0) + \frac{\sigma_0}{\eta}$$

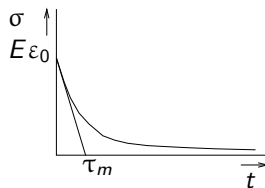
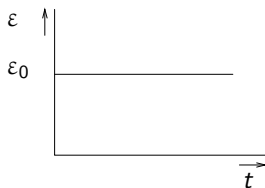
$$\varepsilon(t) = \frac{\sigma_0}{E} H(t, 0) + \frac{\sigma_0}{\eta} t = \sigma_0 \left[ \frac{1}{\eta} \left( t + \frac{\eta}{E} \right) \right] = \sigma_0 D(t)$$



## Maxwell : strain step excitation

$$\varepsilon(t) = \varepsilon_0 H(t, 0) \quad \rightarrow \quad \dot{\varepsilon}(t) = \varepsilon_0 \delta(t, 0)$$

$$\sigma(t) = \varepsilon_0 E e^{-\frac{E}{\eta} t} = \varepsilon_0 E e^{-\frac{t}{\tau_m}} = \varepsilon_0 E(t)$$



# Maxwell : Boltzmann integrals

creep

$$\begin{aligned}\varepsilon(t) &= \frac{1}{\eta} \int_{\tau=-\infty}^t \left\{ (t - \tau) + \frac{\eta}{E} \right\} \dot{\sigma}(\tau) d\tau \\ &= \int_{\tau=-\infty}^t D(t - \tau) \dot{\sigma}(\tau) d\tau\end{aligned}$$

relaxation

$$\begin{aligned}\sigma(t) &= \int_{\tau=-\infty}^t \left\{ E e^{-\frac{E}{\eta}(t-\tau)} \right\} \dot{\varepsilon}(\tau) d\tau \\ &= \int_{\tau=-\infty}^t E(t - \tau) \dot{\varepsilon}(\tau) d\tau\end{aligned}$$

## Maxwell : harmonic stress excitation

$$\sigma(t) = \sigma_0 \sin(\omega t) \quad \rightarrow \quad \dot{\sigma}(t) = \sigma_0 \omega \cos(\omega t)$$

strain response

$$\begin{aligned}\dot{\varepsilon}(t) &= \frac{1}{E} \sigma_0 \omega \cos(\omega t) + \frac{1}{\eta} \sigma_0 \sin(\omega t) \\ \varepsilon(t) &= \sigma_0 \left[ \frac{1}{E} \right] \sin(\omega t) - \sigma_0 \left[ \frac{1}{\eta \omega} \right] \cos(\omega t) \\ &= \varepsilon_P(t) \qquad \qquad \qquad \varepsilon_H \text{ damps out} \\ &= \sigma_0 D' \sin(\omega t) - \sigma_0 D'' \cos(\omega t)\end{aligned}$$

dynamic quantities

$$\begin{aligned}D' &= \frac{1}{E} \quad ; \quad D'' = \frac{1}{\eta \omega} \\ \delta &= \arctan \left( \frac{D''}{D'} \right) = \arctan \left( \frac{E}{\eta \omega} \right)\end{aligned}$$

# Maxwell : harmonic strain excitation

$$\varepsilon(t) = \varepsilon_0 \sin(\omega t) \quad \rightarrow \quad \dot{\varepsilon}(t) = \varepsilon_0 \omega \cos(\omega t)$$

stress response

$$\begin{aligned}\sigma(t) &= \int_{\tau=-\infty}^t E(t-\tau) \dot{\varepsilon}(\tau) d\tau = E \varepsilon_0 \omega e^{-\frac{E}{\eta}t} \int_{\tau=0}^t e^{\frac{E}{\eta}\tau} \cos(\omega\tau) d\tau \\&= \left[ \frac{E \varepsilon_0 \omega}{(\frac{E}{\eta})^2 + \omega^2} \frac{E}{\eta} \right] e^{-\frac{E}{\eta}t} + \\&\quad \left[ \frac{E \varepsilon_0 \omega}{(\frac{E}{\eta})^2 + \omega^2} \omega \right] \sin(\omega t) + \left[ \frac{E \varepsilon_0 \omega}{(\frac{E}{\eta})^2 + \omega^2} \frac{E}{\eta} \right] \cos(\omega t) \\&= \varepsilon_0 \left[ \frac{E \omega}{(\frac{E}{\eta})^2 + \omega^2} \omega \right] \sin(\omega t) + \varepsilon_0 \left[ \frac{E \omega}{(\frac{E}{\eta})^2 + \omega^2} \frac{E}{\eta} \right] \cos(\omega t) \quad \text{for } t \geq 0 \\&= \varepsilon_0 \left[ \frac{E \omega^2 \tau_m^2}{1 + \omega^2 \tau_m^2} \right] \sin(\omega t) + \varepsilon_0 \left[ \frac{E \omega \tau_m}{1 + \omega^2 \tau_m^2} \right] \cos(\omega t) \\&= \varepsilon_0 E' \sin(\omega t) + \varepsilon_0 E'' \cos(\omega t)\end{aligned}$$

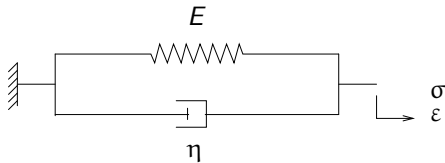


# Maxwell : harmonic strain excitation

dynamic quantities

$$E' = \frac{E\omega^2}{(\frac{E}{\eta})^2 + \omega^2} \quad ; \quad E'' = \frac{E\omega(\frac{E}{\eta})}{(\frac{E}{\eta})^2 + \omega^2} \quad ; \quad \tan(\delta) = \frac{E''}{E'} = \frac{1}{\omega\tau_m}$$

# Kelvin-Voigt model



$$\sigma = \sigma_E + \sigma_\eta = E\epsilon + \eta\dot{\epsilon}$$

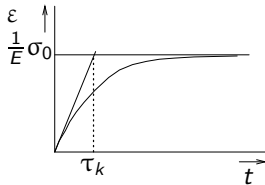
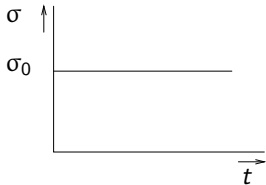
## Kelvin-Voigt : stress step excitation

$$\sigma(t) = \sigma_0 H(t, 0) \quad \rightarrow \quad \dot{\sigma}(t) = \sigma_0 \delta(t, 0)$$

$$\eta \dot{\varepsilon}(t) + E \varepsilon(t) = \sigma(t) = \sigma_0 H(t, 0)$$

$$\left. \begin{aligned} \varepsilon(t) &= \varepsilon_H(t) + \varepsilon_P = C e^{-\frac{E}{\eta} t} + \frac{\sigma_0}{E} \\ \varepsilon(t=0) &= 0 \end{aligned} \right\} \rightarrow C = -\frac{\sigma_0}{E}$$

$$\varepsilon(t) = \frac{\sigma_0}{E} \left[ 1 - e^{-\frac{E}{\eta} t} \right] = \sigma_0 D(t)$$



## Kelvin-Voigt : strain step excitation

$$\varepsilon(t) = \varepsilon_0 H(t, 0) \quad \rightarrow \quad \dot{\varepsilon}(t) = \varepsilon_0 \delta(t, 0)$$

$$\sigma(t) = E\varepsilon(t) + \eta\dot{\varepsilon}(t)$$

$$\sigma(t) = E\varepsilon_0 + \eta\varepsilon_0\delta(t, 0) = \varepsilon_0 [E + \eta\delta(t, 0)] = \infty$$

## Kelvin-Voigt : Boltzmann integral

$$\begin{aligned}\varepsilon(t) &= \frac{1}{E} \int_{\tau=-\infty}^t \left\{ 1 - e^{-\frac{E}{\eta}(t-\tau)} \right\} \dot{\sigma}(\tau) d\tau \\ &= \int_{\tau=-\infty}^t D(t-\tau) \dot{\sigma}(\tau) d\tau\end{aligned}$$

## Kelvin-Voigt : harmonic stress excitation

$$\sigma(t) = \sigma_0 \sin(\omega t) \quad \rightarrow \quad \dot{\sigma}(t) = \sigma_0 \omega \cos(\omega t)$$

strain response

$$\begin{aligned}\varepsilon(t) &= \int_{\tau=0}^t D(t-\tau) \dot{\sigma}(\tau) d\tau \\ &= \sigma_0 \left[ \frac{1}{\left(\frac{E}{\eta}\right)^2 + \omega^2} \frac{E}{\eta^2} \right] \sin(\omega t) - \sigma_0 \left[ \frac{\omega}{\left(\frac{E}{\eta}\right)^2 + \omega^2} \frac{1}{\eta} \right] \cos(\omega t) \\ &= \sigma_0 \left[ \frac{1}{E(1 + \omega^2 \tau_k^2)} \right] \sin(\omega t) - \sigma_0 \left[ \frac{\omega \tau_k}{E(1 + \omega^2 \tau_k^2)} \right] \cos(\omega t) \\ &= \sigma_0 D' \sin(\omega t) - \sigma_0 D'' \cos(\omega t)\end{aligned}$$

# Kelvin-Voigt : harmonic stress excitation

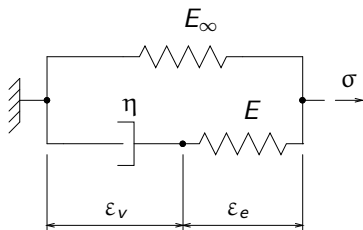
dynamic quantities

$$D'(\omega) = \frac{1}{\left(\frac{E}{\eta}\right)^2 + \omega^2} \frac{E}{\eta^2} = \frac{1}{E(1 + \omega^2\tau_k^2)}$$

$$D''(\omega) = \frac{\omega}{\left(\frac{E}{\eta}\right)^2 + \omega^2} \frac{1}{\eta} = \frac{\omega\tau}{E(1 + \omega^2\tau_k^2)}$$

$$\tan(\delta) = \frac{D''}{D'} = \omega\tau_k \quad \rightarrow \quad \delta = \arctan\left(\frac{\eta\omega}{E}\right)$$

# Standard Solid model

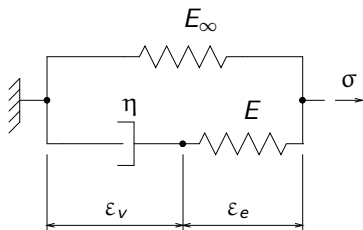


constitutive relations

- $\sigma = \sigma_\infty + \sigma_{ve}$
- $\dot{\varepsilon} = \dot{\varepsilon}_v + \dot{\varepsilon}_e$
- $\dot{\varepsilon}_v = \frac{1}{\eta} \sigma_{ve}$
- $\sigma_{ve} = E \varepsilon_e \rightarrow \dot{\varepsilon}_e = \frac{1}{E} \dot{\sigma}_{ve}$
- $\varepsilon = \frac{1}{E_\infty} \sigma_\infty$



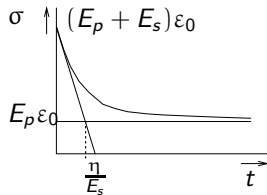
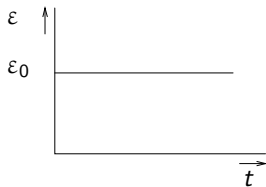
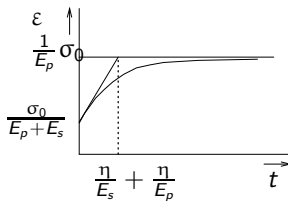
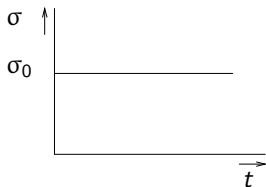
# Standard Solid model



constitutive equation

$$\begin{aligned}\sigma &= \sigma_\infty + \sigma_{ve} = E_\infty \epsilon + \eta \dot{\epsilon}_v \\ &= E_\infty \epsilon + \eta (\dot{\epsilon} - \dot{\epsilon}_e) = E_\infty \epsilon + \eta \dot{\epsilon} - \eta \frac{\dot{\sigma}_{ve}}{E} \\ &= E_\infty \epsilon + \eta \dot{\epsilon} - \frac{\eta}{E} (\dot{\sigma} - E_\infty \dot{\epsilon}) \rightarrow \\ \sigma + \frac{\eta}{E} \dot{\sigma} &= E_\infty \epsilon + \frac{\eta(E + E_\infty)}{E} \dot{\epsilon}\end{aligned}$$

# Standard Solid : step excitations

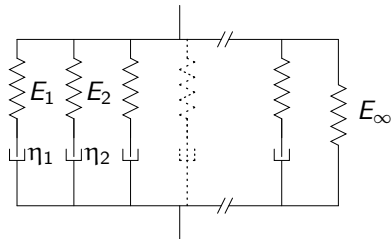


## Standard Solid : Boltzmann integrals

$$\begin{aligned}\varepsilon(t) &= \int_{\tau=-\infty}^t \left\{ \frac{1}{E_{\infty}} - \frac{E}{E_{\infty}(E_{\infty} + E)} e^{-\frac{E_{\infty}E}{\eta(E_{\infty} + E)}(t-\tau)} \right\} \dot{\sigma}(\tau) d\tau \\ &= \int_{\tau=-\infty}^t D(t-\tau) \dot{\sigma}(\tau) d\tau\end{aligned}$$

$$\begin{aligned}\sigma(t) &= \int_{\tau=-\infty}^t \left\{ E_{\infty} + E e^{-\frac{E}{\eta}(t-\tau)} \right\} \dot{\varepsilon}(\tau) d\tau \\ &= \int_{\tau=-\infty}^t E(t-\tau) \dot{\varepsilon}(\tau) d\tau\end{aligned}$$

# Generalized Maxwell model



$$E(t) = E_{\infty} + \sum_i E_i e^{-\frac{t}{\tau_i}} \quad ; \quad \tau_i = \frac{\eta_i}{E_i}$$

equilibrium modulus

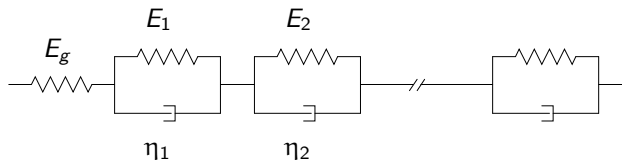
$$E_{\infty} = \lim_{t \rightarrow \infty} E(t)$$

glass modulus

$$E_g = \lim_{t \rightarrow 0} E(t) = E_{\infty} + \sum_i E_i$$

$i = 1 \quad \rightarrow \quad$  Standard Solid Model

# Generalized Kelvin model



$$D(t) = \frac{1}{E_g} + \sum_i \frac{1}{E_i} (1 - e^{-\frac{t}{\tau_i}}) \quad ; \quad \tau_i = \frac{\eta_i}{E_i}$$

$$= D_g + \sum_i D_i (1 - e^{-\frac{t}{\tau_i}})$$

glass compliance

$$D_g = \frac{1}{E_g} = \lim_{t \rightarrow 0} D(t)$$

equilibrium compliance

$$D_\infty = \lim_{t \rightarrow \infty} D(t) = D_g + \sum_i D_i$$

viscoelastic liquid : extra serial dashpot with  
*end viscosity*  $\eta_v \rightarrow D(t) = .. + \frac{1}{\eta_v} t$

# NUMERICAL IMPLEMENTATION

[back to index](#)

# Stress update

stress must be updated  $\rightarrow$   
Boltzmann integral must be calculated

we assume "Generalized Maxwell model"

# Stress relaxation

$$\left. \begin{aligned} \sigma(t) &= \int_{\tau=0}^t E(t-\tau) \dot{\varepsilon}(\tau) d\tau \\ E(t) &= E_{\infty} + \sum_{i=1}^N E_i e^{-\frac{t}{\tau_i}} \end{aligned} \right\} \rightarrow$$

$$\begin{aligned} \sigma(t) &= \int_{\tau=0}^t \left[ E_{\infty} + \sum_{i=1}^N E_i e^{-\frac{t-\tau}{\tau_i}} \right] \dot{\varepsilon}(\tau) d\tau \\ &= E_{\infty} \varepsilon(t) + \sum_{i=1}^N \int_{\tau=0}^t E_i e^{-\frac{t-\tau}{\tau_i}} \dot{\varepsilon}(\tau) d\tau \end{aligned}$$

To evaluate the integral, the time is discretized



# Time discretization

$$\begin{aligned}\sigma(t) &= E_{\infty} \varepsilon(t) + \sum_{i=1}^N \int_{\tau=0}^t E_i e^{-\frac{t-\tau}{\tau_i}} \dot{\varepsilon}(\tau) d\tau \\ &= E_{\infty} \varepsilon(t) + \sum_{i=1}^N \left[ e^{-\frac{\Delta t}{\tau_i}} \int_{\tau=0}^{t_n} E_i e^{-\frac{t_n-\tau}{\tau_i}} \dot{\varepsilon}(\tau) d\tau + E_i \int_{\tau=t_n}^t e^{-\frac{t-\tau}{\tau_i}} \dot{\varepsilon}(\tau) d\tau \right]\end{aligned}$$

Only the integral over the current increment has to be calculated  
This can be done analytically after assumption for strain rate :

incremental strain rate is constant

## Linear incremental strain

$$\begin{aligned} & \int_{\tau=t_n}^t e^{-\frac{t-\tau}{\tau_i}} \dot{\epsilon}(\tau) d\tau \\ &= \frac{\Delta \epsilon}{\Delta t} \int_{\tau=t_n}^t e^{-\frac{t-\tau}{\tau_i}} d\tau \\ &= \frac{\Delta \epsilon}{\Delta t} \tau_i \left( 1 - e^{-\frac{\Delta t}{\tau_i}} \right) \end{aligned}$$

# Stress

$$\begin{aligned}\sigma(t) &= E_{\infty} \varepsilon(t) + \sum_{i=1}^N \sigma_i(t) \\ &= E_{\infty} \varepsilon(t) + \sum_{i=1}^N \left[ e^{-\frac{\Delta t}{\tau_i}} \int_{\tau=0}^{t_n} E_i e^{-\frac{t_n-\tau}{\tau_i}} \dot{\varepsilon}(\tau) d\tau + \right. \\ &\quad \left. E_i \tau_i \left( 1 - e^{-\frac{\Delta t}{\tau_i}} \right) \frac{\Delta \varepsilon}{\Delta t} \right]\end{aligned}$$

$$\sigma(t) = E_{\infty} \varepsilon(t) + \sum_{i=1}^N \left[ e^{-\frac{\Delta t}{\tau_i}} \sigma_i(t_n) + E_i p_i \Delta \varepsilon \right]$$

$$\text{with} \quad p_i = \frac{\tau_i}{\Delta t} \left( 1 - e^{-\frac{\Delta t}{\tau_i}} \right)$$

# Stiffness

$$\sigma(t) = E_{\infty} \varepsilon(t) + \sum_{i=1}^N \left[ e^{-\frac{\Delta t}{\tau_i}} \sigma_i(t_n) + E_i p_i \Delta \varepsilon \right] \rightarrow$$

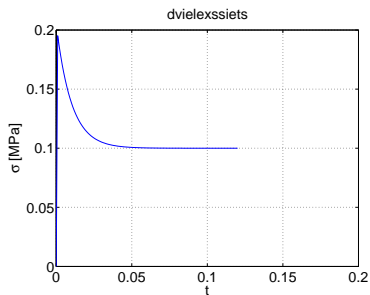
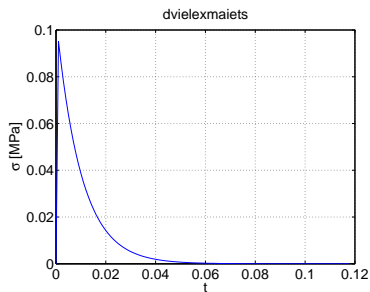
$$\frac{\partial \sigma}{\partial \lambda} = C_{\lambda} = C_{\varepsilon} = E_{\infty} + \sum_{i=1}^N E_i p_i$$

# Implementation

`tr2dviel.m`

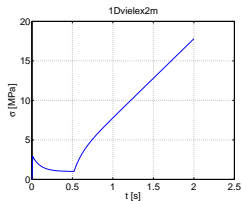
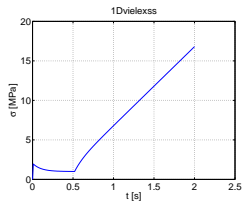
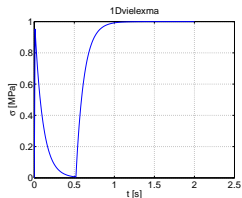
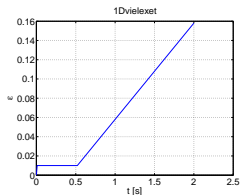
# Strain step

Maxwell	$E_{\infty} = 0$	$E_1 = 1$	$\tau_1 = 0.01$
Standard-Solid	$E_{\infty} = 1$	$E_1 = 1$	$\tau_1 = 0.01$



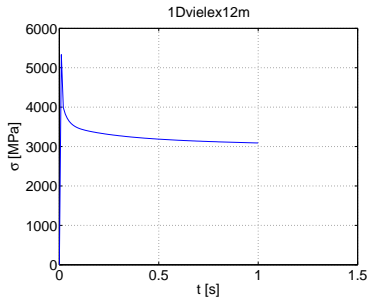
# Linear viscoelastic models

	$E_\infty$	$E_1$	$\tau_1$	$E_2$	$\tau_2$	$\nu$
Maxwell	0	100	0.1	0	0	0
Kelvin-Voigt	100	$10^{10}$	$10^{-12}$	0	0	0
Standard-Solid	100	100	0.1	0	0	0
2-mode	100	100	0.1	100	0.1	0



# Multi-mode model response

	$E$ [MPa]	$\tau$ [s]		$E$ [MPa]	$\tau$ [s]
1	3.0e6	3.1e-8	2	1.4e6	3.0e-7
3	3.9e6	3.0e-6	4	5.4e6	2.9e-5
5	1.3e6	2.8e-4	6	2.3e5	2.7e-3
7	7.6e4	2.6e-2	8	3.7e4	2.5e-1
9	3.3e4	2.5e+0	10	1.7e4	2.4e+1
11	8.0e3	2.3e+2	12	1.2e4	2.2e+3

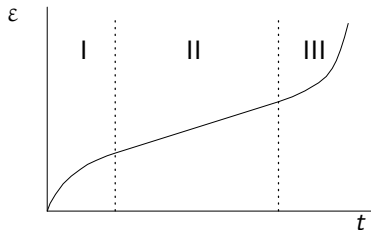




## CREEP

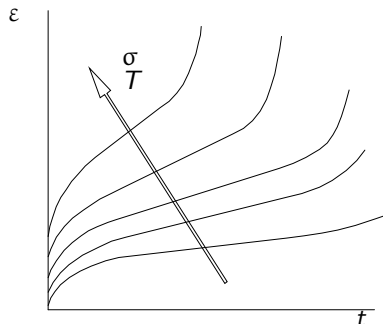
[back to index](#)

# Creep behavior



- primary / stage I / transient creep (delayed elastic effect)
- secondary / steady-state / stage II creep (viscous flow)
- tertiary / stage III / accelerating creep

# Creep strain rate



general model

$$\dot{\epsilon}_c = A f_{\sigma}(\sigma) f_{\epsilon}(\epsilon_c) f_T(T) f_t(t)$$

power law model

$$\dot{\epsilon}_c = A \sigma^m \epsilon_c^n T^p (qt^{q-1})$$

# Primary creep

- $T < 0.4T_m$
- dislocation coalescence  $\rightarrow$  entanglement / pile-up  $\rightarrow$
- hardening
- time-dependent plasticity

$$\dot{\epsilon}_c = C(\sigma) \exp[-\alpha(\sigma)t]$$

## Secondary creep

- $0.5T_m < T < 0.6T_m$
- vacancy movement (self diffusion)

$$\dot{\epsilon}_c = A \exp\left(-\frac{Q_c}{RT}\right) \left(\frac{\sigma}{E}\right)^n \quad (n \approx 5)$$

power-law-breakdown model

$$\dot{\epsilon}_c = A \exp\left(-\frac{Q_c}{RT}\right) \left(\sinh\left(\alpha \frac{\sigma}{E}\right)\right)^5$$

# Tertiary creep

- $0.6T_m < T < 0.8T_m$
- grain boundary sliding  $\rightarrow$  void initiation/coalescence  $\rightarrow$  inter granular cracks
- diffusional flow
- modelled with damage mechanics

tertiary creep damage model (Kachanov & Rabotnov)

$$\frac{\dot{\epsilon}_c}{\dot{\epsilon}_{c0}} = \frac{(\sigma/\sigma_0)^n}{(1-\omega)^m} \quad ; \quad \frac{\dot{\omega}}{\dot{\omega}_0} = \frac{(\sigma/\sigma_0)^\nu}{(1-\omega)^\mu} \quad (n \geq \nu)$$

# Stress functions

Norton; Bailey (1929)

$$\dot{\epsilon}_c = K \sigma^n$$

Hooke-Norton

$$\dot{\epsilon}_c = \frac{\dot{\sigma}}{E} + K \sigma^n$$

Johnson et.al. (1963)

$$\dot{\epsilon}_c = D_1 \sigma^{n_1} + D_2 \sigma^{n_2}$$

Dorn (1955)

$$\dot{\epsilon}_c = B \exp(\beta \sigma)$$

Soderberg (1936)

$$\dot{\epsilon}_c = B \left[ \exp \left( \frac{\sigma}{\sigma_0} \right) - 1 \right]$$

Prandtl (1928)

$$\dot{\epsilon}_c = A \sinh \left( \frac{\sigma}{\sigma_0} \right)$$

Garofalo (1965)

$$\dot{\epsilon}_c = A \left[ \sinh \left( \frac{\sigma}{\sigma_0} \right) \right]^n$$

Lemaitre, Chaboche (1985)

$$\dot{\epsilon}_c = \left( \frac{\sigma}{\lambda_0} \right)^{N_0} \exp(\alpha \sigma^{N_0+1})$$

# Temperature functions

Kauzmann (1941)

$$\dot{\varepsilon}_c = A \exp \left( - \frac{\Delta H - \gamma \sigma}{RT} \right)$$

Lifszic (1963)

$$\dot{\varepsilon}_c = \frac{\sigma}{T} \exp \left( - \frac{\Delta H}{RT} \right)$$

Dorn, Tietz (1949/55)

$$\varepsilon_c = f \left( t \exp \left[ - \frac{\Delta H}{RT} \right] \right) f_{\sigma}(\sigma)$$

Penny, Marriott (1971)

$$\varepsilon_c = \left( t \exp \left[ - \frac{\Delta H}{RT} \right] \right)^n f_{\sigma}(\sigma)$$

Boyle, Spence (1983)

$$\varepsilon_c = C \exp \left( - \frac{\Delta H}{RT} \right) t^m \sigma^n$$



# Time functions

Andrade (1910)

$$\varepsilon_c = \ln \left( 1 + \beta t^{\frac{1}{3}} \right) + kt$$

Andrade (small  $\varepsilon$ )

$$\varepsilon_c = \beta t^{\frac{1}{3}} + kt \approx \beta t^{\frac{1}{3}}$$

Bailey (1935)

$$\varepsilon_c = Ft^n$$

Graham, Walles (1955)

$$\varepsilon_c = \sum_{j=1}^M a_j t^{m_j}$$

McVetty (1934)

$$\varepsilon_c = G (1 - \exp(-qt)) + Ht$$

Findley et.al. (1944)

$$\varepsilon_c = \varepsilon_1 + \varepsilon_2 t^n \quad (n < 1)$$

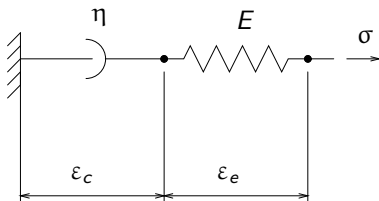
Pugh (1975)

$$\varepsilon_c = \frac{a_1 t}{1 + b_1 t} + \frac{a_2 t}{1 + b_2 t} + \dot{\varepsilon}_m t$$

Garofalo

$$\varepsilon_c = \varepsilon_t (1 - e^{-rt}) + \dot{\varepsilon}_s t$$

# Creep model



constitutive relations

- $\dot{\varepsilon} = \dot{\varepsilon}_e + \dot{\varepsilon}_c$
- $\sigma = E\varepsilon_e \rightarrow \dot{\varepsilon}_e = \frac{1}{E} \dot{\sigma}$
- $\dot{\varepsilon}_c = A f_{\sigma}(\sigma) f_{\varepsilon_c}(\varepsilon_c) f_T(T) f_t(t) = f(\sigma, \varepsilon_c, T, t)$

constitutive equation

$$\dot{\sigma} = E\dot{\varepsilon}_e = E\dot{\varepsilon} - E\dot{\varepsilon}_c = E\dot{\varepsilon} - Ef(\sigma, \varepsilon_c, T, t)$$

# NUMERICAL IMPLEMENTATION

[back to index](#)

# Stress update

$$\dot{\sigma} = E\dot{\varepsilon} - Ef(\sigma, \varepsilon_c, T, t)$$

$$\Delta\sigma = E\Delta\varepsilon - \Delta t Ef(\sigma, \varepsilon_c, T, t)$$

$$\sigma - \sigma_n = E(\varepsilon - \varepsilon_n) - \Delta t Ef(\sigma, \varepsilon_c, T, t)$$

- implicit procedure
- explicit procedure

# Implicit stress update

$$\sigma - \sigma_n = E(\varepsilon - \varepsilon_n) - \Delta t E f(\sigma, \varepsilon_c, T, t)$$

$$\begin{aligned}\sigma^* + \delta\sigma - \sigma_n &= E(\varepsilon - \varepsilon_n) - \Delta t E (f^* + \delta f) = E(\varepsilon - \varepsilon_n) - \Delta t E f^* - \Delta t E \delta f \\ &= E(\varepsilon - \varepsilon_n) - \Delta t E f^* - \Delta t E \frac{\partial f}{\partial \sigma} \delta\sigma \quad \rightarrow\end{aligned}$$

$$\left[ 1 + \Delta t E \frac{\partial f}{\partial \sigma} \right] \delta\sigma = -\sigma^* + \sigma_n + E(\varepsilon - \varepsilon_n) - \Delta t E f^*$$

# Explicit stress update

$$\sigma = \sigma_n + E(\varepsilon - \varepsilon_n) - \Delta t Ef(\sigma_n, \varepsilon_{c_n}, T_n, t_n)$$

# Stiffness

implicit

$$\sigma - \sigma_n - E\varepsilon + E\varepsilon_n + \Delta t Ef(\sigma, \varepsilon_c, T, t) = 0$$

$$\delta\sigma + \Delta t E \left. \frac{\partial f}{\partial \sigma} \right|^* \delta\sigma - E\delta\varepsilon = 0$$

$$C_\varepsilon = \left( 1 + \Delta t E \left. \frac{\partial f}{\partial \sigma} \right|^* \right)^{-1} E$$

explicit

$$\sigma - \sigma_n - E\varepsilon + E\varepsilon_n + \Delta t Ef(\sigma_n, \varepsilon_{c_n}, T_n, t_n) = 0$$

$$\delta\sigma = E\delta\varepsilon \quad \rightarrow \quad C_\varepsilon = E$$

# Implementation

`tr2delvi.m`



# Creep versus viscoelasticity

Maxwell model ( $E, \eta$ )

$$\varepsilon = \varepsilon_e + \varepsilon_c \quad ; \quad E(t) = E e^{t/\tau} \quad ; \quad \tau = \frac{\eta}{E}$$

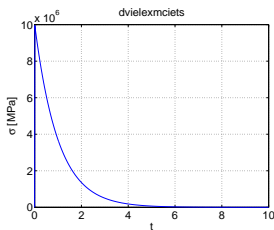
$$\dot{\varepsilon}_c = \frac{\sigma}{\eta} \quad ; \quad \varepsilon_e = \frac{\sigma}{E}$$

Norton model ( $A, m$ )

$$\varepsilon = \varepsilon_e + \varepsilon_c \quad ; \quad \dot{\varepsilon}_c = f(\sigma, \varepsilon_c, T, t)$$

$$\dot{\varepsilon}_c = A \sigma^m \quad ; \quad \varepsilon_e = \frac{\sigma}{E}$$

Maxwell	$E = 10^9$	$\eta = 10^9$	$\tau = 1$
Norton	$E = 10^9$	$A = \frac{1}{\eta} = 10^{-9}$	$m = 1$



## General creep model for SnAg-solder

$$\varepsilon_c(t) = \varepsilon_0 + A(\sigma) \left[ 1 - e^{-\alpha(\sigma, T)t} \right] + B(\sigma, T) \left[ e^{\alpha(\sigma, T)t} - 1 \right]$$

$$\alpha(\sigma, T) = c_1 [\sinh(\beta\sigma)]^{n_1} e^{-\frac{Q_1}{RT}}$$

$$A(\sigma) = c_2 \sigma^{n_2}$$

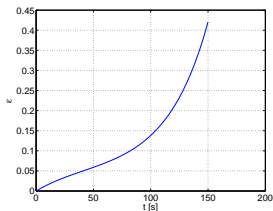
$$B(\sigma, T) = c_3 \sigma^{n_3} e^{-\frac{Q_2}{RT}}$$

# General creep model for SnAg-solder

$$\dot{\varepsilon}_c = A\alpha e^{-\alpha t} + B\alpha e^{\alpha t} ; \quad \dot{\varepsilon}_{c,i} = \dot{\varepsilon}_c(t=0) = \alpha(A+B) ; \quad t_m = \frac{1}{2\alpha} \ln\left(\frac{A}{B}\right)$$

$$\dot{\varepsilon}_{c,m} = \dot{\varepsilon}_c(t=t_m) = 2\alpha\sqrt{AB} \quad ; \quad \varepsilon_{c,m} = \varepsilon_c(t=t_m) = \varepsilon_0 + A - B$$

$\varepsilon_0$	0		
$c_1$	$1.73 \times 10^5$	$n_1$	4.66
$\beta$	0.095	$Q_1$	70
$c_2$	$2.06 \times 10^{-3}$	$n_2$	1.1
$c_3$	$9.65 \times 10^{-4}$	$n_3$	2.38
$Q_2$	17.8		

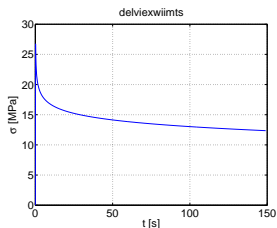


# Special creep model for SnAg-solder

Wiese (2005) : 2-term model Sn4Ag0.5Cu

$$\dot{\epsilon}_c = A_1 \sigma^{m_1} e^{e_1/T} + A_2 \sigma^{m_2} e^{e_2/T}$$

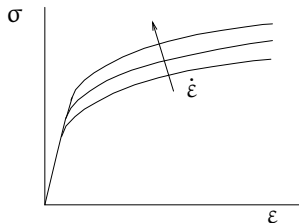
$E = 59.533 - 66.667 T$		
$A_1 = 4.10^{-7}$	$m_1 = 3$	$e_1 = -3223$
$A_2 = 1.10^{-12}$	$m_2 = 12$	$e_2 = -7348$



# VISCOPLASTIC

[back to index](#)

# Viscoplastic material behavior



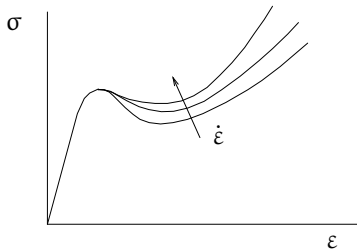
viscoelastic (rate effects)

elastoplastic (permanent deformation)



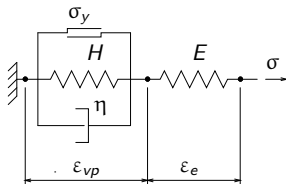
→ viscoplastic

# Softening



- polymers

# Viscoplastic (Perzyna) model



constitutive relations

- $f = \bar{\sigma} - \sigma_y$  with  $f < 0 \rightarrow$  elastic  
 $f \geq 0 \rightarrow$  viscoplastic
- $\sigma_y = \sigma_y(\sigma_{y0}, \bar{\epsilon}_{vp})$
- $\dot{\epsilon} = \dot{\epsilon}_e + \dot{\epsilon}_{vp}$
- $\sigma = E \epsilon_e \rightarrow \dot{\epsilon}_e = \frac{1}{E} \dot{\sigma}$
- $\dot{\epsilon}_{vp} = \dot{\lambda} \frac{\partial f}{\partial \sigma} = \dot{\lambda} \frac{\sigma}{\bar{\sigma}} ; \quad \dot{\epsilon}_{vp} = |\dot{\epsilon}_{vp}|$
- $\bar{\epsilon}_{vp} = \int_{\tau=0}^t \dot{\epsilon}_{vp} d\tau$
- $\dot{\lambda} = \gamma \phi(f) = \gamma (f/\sigma_{y0})^N$



# Hardening laws

$$\sigma_y = \sigma_{y0} + H\bar{\epsilon}_{vp} + a\bar{\epsilon}_{vp}^2 + b\bar{\epsilon}_{vp}^3 + c\bar{\epsilon}_{vp}^4 + d\bar{\epsilon}_{vp}^7$$

- parameters fitted with compression tests to prevent instability
- 7th-order polynomial is used for polycarbonate

# Constitutive equations

$$\begin{cases} \dot{\sigma} = E\dot{\varepsilon}_e = E(\dot{\varepsilon} - \dot{\varepsilon}_{vp}) = E\{\dot{\varepsilon} - \dot{\lambda} \left(\frac{\sigma}{\bar{\sigma}}\right)\} \\ \dot{\lambda} = \gamma\phi \end{cases}$$
$$\begin{cases} \Delta\sigma = E\Delta\varepsilon - E\Delta\lambda \left(\frac{\sigma}{\bar{\sigma}}\right) \\ \Delta\lambda = \gamma\phi\Delta t \end{cases}$$
$$\begin{cases} \sigma - \sigma_n = E\varepsilon - E\varepsilon_n - E(\lambda - \lambda_n) \left(\frac{\sigma}{\bar{\sigma}}\right) \\ \lambda - \lambda_n = \gamma\phi\Delta t \end{cases}$$

- $f > 0$  is allowed  $\rightarrow$  "overstress model"  
(no consistency equation)
- viscoplastic multiplier  $\lambda$  cannot be eliminated

# NUMERICAL IMPLEMENTATION

[back to index](#)

# Elastic stress predictor

$$\sigma_e = \sigma_n + E(\varepsilon - \varepsilon_n)$$

- $f = \bar{\sigma}_e - \sigma_{y_n} \leq 0 \quad \rightarrow \quad \text{elastic increment}$
- $f = \bar{\sigma}_e - \sigma_{y_n} > 0 \quad \rightarrow \quad \text{elastoviscoplastic increment}$

# Elastic increment

$$\sigma(t_{n+1}) = \sigma_e$$

$$\bar{\varepsilon}_{vp}(t_{n+1}) = \bar{\varepsilon}_{vp}(t_n) = \bar{\varepsilon}_{vp_n}$$

$$\sigma_y(t_{n+1}) = \sigma_y(t_n) = \sigma_{y_n}$$

# Elastoviscoplastic increment

$$\begin{cases} \Delta\sigma = E\Delta\varepsilon - E\Delta\lambda \left( \frac{\sigma}{\bar{\sigma}} \right) \\ \Delta\lambda = \gamma\phi\Delta t \end{cases}$$
$$\begin{cases} \sigma - \sigma_n = E\varepsilon - E\varepsilon_n - E(\lambda - \lambda_n) \left( \frac{\sigma}{\bar{\sigma}} \right) \\ \lambda - \lambda_n = \Delta t\gamma\phi \end{cases}$$

- implicit
- explicit

# Implicit stress update

$$\begin{cases} \sigma - \sigma_n = E\varepsilon - E\varepsilon_n - E(\lambda - \lambda_n) \left( \frac{\sigma}{\bar{\sigma}} \right) \\ \lambda - \lambda_n = \Delta t \gamma \phi \end{cases}$$
$$\begin{cases} \sigma^* + \delta\sigma - \sigma_n = E\varepsilon - E\varepsilon_n - E(\lambda^* + \delta\lambda - \lambda_n) \left\{ \left( \frac{\sigma}{\bar{\sigma}} \right)^* + \delta \left( \frac{\sigma}{\bar{\sigma}} \right) \right\} \\ \lambda^* + \delta\lambda - \lambda_n = \Delta t \gamma (\phi^* + \delta\phi) \end{cases}$$

linearization and reorganization

$$\begin{cases} \delta\sigma + \left[ E \left( \frac{\sigma}{\bar{\sigma}} \right)^* \right] \delta\lambda \\ \quad = -\sigma^* + \sigma_n + E\varepsilon - E\varepsilon_n - E(\lambda^* - \lambda_n) \left( \frac{\sigma}{\bar{\sigma}} \right)^* \\ \left[ -\Delta t \gamma \frac{\partial \phi}{\partial \sigma} \right] \delta\sigma + \left[ 1 - \Delta t \gamma \frac{\partial \phi}{\partial \lambda} \right] \delta\lambda \\ \quad = -\lambda^* + \lambda_n + \Delta t \gamma \phi^* \end{cases}$$

# Implicit stress update

$$\frac{\partial \phi}{\partial \lambda} = \frac{d\phi}{df} \frac{df}{d\sigma_y} \frac{d\sigma_y}{d\bar{\varepsilon}_{vp}} \frac{d\bar{\varepsilon}_{vp}}{d\lambda} = \frac{d\phi}{df} (-1) H \left( \frac{\sigma}{\bar{\sigma}} \right)^* = - \frac{d\phi}{df} H \left( \frac{\sigma}{\bar{\sigma}} \right)^*$$

$$\frac{\partial \phi}{\partial \sigma} = \frac{d\phi}{df} \frac{df}{d\sigma} = \frac{d\phi}{df} \left( \frac{\sigma}{\bar{\sigma}} \right)^*$$

$$\frac{d\phi}{df} = N \left( \frac{f}{\sigma_{y0}} \right)^{N-1} \frac{1}{\sigma_{y0}}$$



## Explicit stress update

$$\begin{cases} \sigma - \sigma_n = E\varepsilon - E\varepsilon_n - E(\lambda - \lambda_n) \left( \frac{\sigma}{\bar{\sigma}} \right) \\ \lambda - \lambda_n = \Delta t \gamma \phi \end{cases}$$
$$\begin{cases} \sigma - \sigma_n = E\varepsilon - E\varepsilon_n - E(\lambda - \lambda_n) \left( \frac{\sigma_n}{\bar{\sigma}_n} \right) \\ \lambda - \lambda_n = \Delta t \gamma \phi_n \end{cases}$$
$$\begin{cases} \sigma + E \left( \frac{\sigma_n}{\bar{\sigma}_n} \right) \lambda = \sigma_n + E\varepsilon - E\varepsilon_n + E\lambda_n \left( \frac{\sigma_n}{\bar{\sigma}_n} \right) \\ \lambda = \lambda_n + \Delta t \gamma \phi_n \left( \frac{\sigma_n}{\bar{\sigma}_n} \right) \end{cases}$$

# Stiffness : implicit

$$\left. \begin{aligned} \sigma - \sigma_n &= E(\varepsilon - \varepsilon_n) - E(\lambda - \lambda_n) \left( \frac{\sigma}{\bar{\sigma}} \right) \\ \lambda - \lambda_n &= \Delta t \gamma \phi \end{aligned} \right\}$$

$$\left. \begin{aligned} \delta \sigma &= E \delta \varepsilon - E \delta \lambda \left( \frac{\sigma}{\bar{\sigma}} \right) - E(\lambda - \lambda_n) \left( \frac{1}{\bar{\sigma}} \right) \delta \sigma \\ \delta \lambda &= \Delta t \gamma \delta \phi = \Delta t \gamma \frac{\partial \phi}{\partial \sigma} \delta \sigma + \Delta t \gamma \frac{\partial \phi}{\partial \lambda} \delta \lambda \end{aligned} \right\}$$

$$\delta \sigma = E \delta \varepsilon - E \left( \frac{\sigma}{\bar{\sigma}} \right) \frac{\gamma \Delta t \frac{\partial \phi}{\partial \sigma}}{1 - \gamma \Delta t \frac{\partial \phi}{\partial \lambda}} \delta \sigma - E(\lambda - \lambda_n) \left( \frac{1}{\bar{\sigma}} \right) \delta \sigma$$

$$C_\varepsilon = \frac{E \left\{ 1 - \gamma \Delta t \frac{\partial \phi}{\partial \lambda} \right\}}{\left\{ 1 - \gamma \Delta t \frac{\partial \phi}{\partial \lambda} \right\} + E \left( \frac{\sigma}{\bar{\sigma}} \right) \gamma \Delta t \frac{\partial \phi}{\partial \sigma} + E(\lambda - \lambda_n) \frac{1}{\bar{\sigma}} \left\{ 1 - \gamma \Delta t \frac{\partial \phi}{\partial \lambda} \right\}}$$

## Stiffness : explicit

$$\left\{ \begin{array}{l} \sigma - \sigma_n = E\varepsilon - E\varepsilon_n - E(\lambda - \lambda_n) \left( \frac{\sigma_n}{\bar{\sigma}_n} \right) \\ \lambda - \lambda_n = \Delta t \gamma \phi_n \end{array} \right.$$

$$\left\{ \begin{array}{l} \delta\sigma = E\delta\varepsilon - E\delta\lambda \left( \frac{\sigma_n}{\bar{\sigma}_n} \right) \\ \delta\lambda = 0 \end{array} \right.$$

$$C_\varepsilon = E$$

# Implementation

`tr2dperz.m`

## Prescribed constant strain rates

$$\Delta l(t) = u(t) = u_0 f(t) \quad \rightarrow \quad \lambda(t) = 1 + \frac{u_0}{l_0} f(t) \quad \rightarrow \quad f(t) = \frac{l_0}{u_0} (\lambda(t) - 1)$$

### linear strain

$$\dot{\epsilon}_l = \dot{\lambda} = c \quad \rightarrow \quad \lambda(t) = ct + 1 \quad \rightarrow \quad f(t) = \frac{l_0}{u_0} c t$$

$$\lambda_e = \lambda(t_e) = ct_e + 1 \quad \rightarrow \quad t_e = \frac{1}{c} (\lambda_e - 1)$$

### logarithmic strain

$$\dot{\epsilon}_{ln} = \frac{\dot{\lambda}}{\lambda} = c \quad \rightarrow \quad \ln(\lambda) = ct \quad \rightarrow \quad \lambda(t) = e^{ct} \quad \rightarrow \quad f(t) = \frac{l_0}{u_0} (e^{ct} - 1)$$

$$\lambda_e = \lambda(t_e) = e^{ct_e} \quad \rightarrow \quad t_e = \frac{1}{c} \ln(\lambda_e)$$

# Tensile test at various strain rates



initial length	$l_0$	100	mm
initial cross-sectional area	$A_0$	10	mm <sup>2</sup>

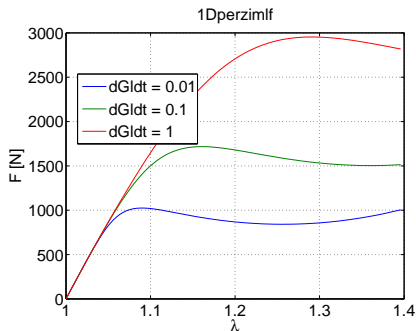
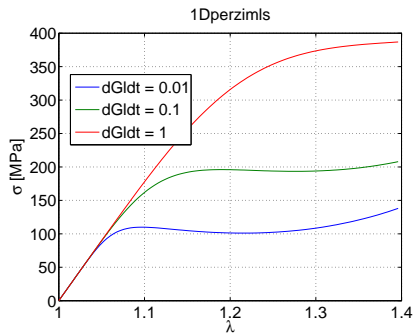
$$\sigma_y = \sigma_{y0} + H\bar{\epsilon}_{vp} + a\bar{\epsilon}_{vp}^2 + b\bar{\epsilon}_{vp}^3 + c\bar{\epsilon}_{vp}^4 + d\bar{\epsilon}_{vp}^7$$

$E$	1800	MPa	$\nu$	0.37	-
$\sigma_{y0}$	37	MPa	$H$	-200	MPa
$\gamma$	0.001	1/s	$N$	3	-
$a$	500	MPa	$b$	700	MPa
$c$	800	MPa	$d$	30000	MPa

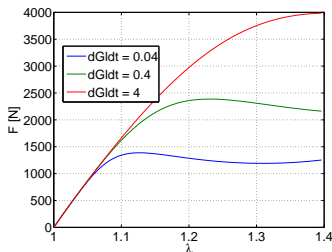
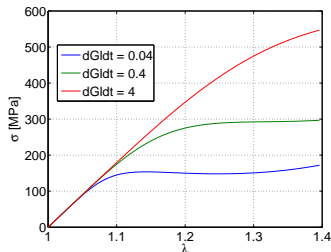
$$\dot{\epsilon}_I = \{0.01, 0.1, 1\}$$

# Tensile test at various strain rates

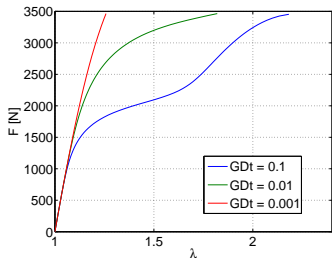
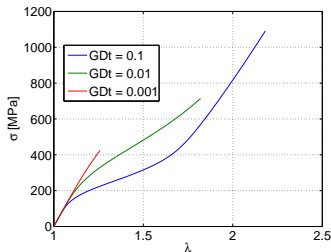
NB: linear strain is used :  $\dot{\epsilon} = \dot{\lambda} = d\lambda/dt$



# Tensile test at various time steps



Prescribed elongation;  $nic = 100 \rightarrow t_e = \{10, 1, 0.1\}$ ;



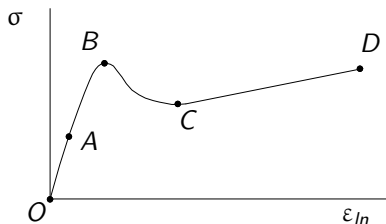
Prescribed force;  $F_{max} = 3500$ ;  $nic = 100 \rightarrow t_e = \{10, 1, 0.1\}$ ;



# NONLINEAR VISCOELASTIC

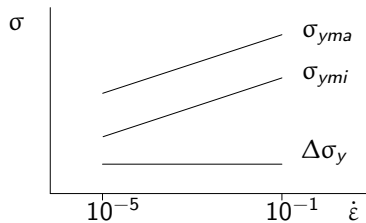
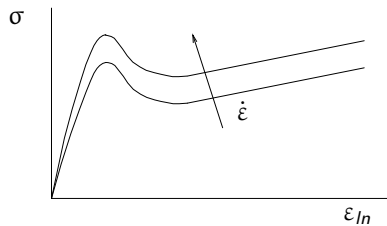
[back to index](#)

# Nonlinear viscoelastic material behavior

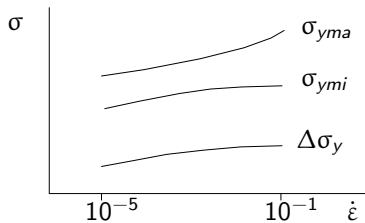
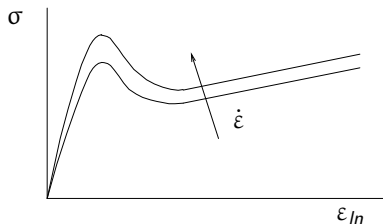


- compression test of polymers →
- OA : linear viscoelastic
- AB : nonlinear viscoelastic
- B : " (maximum) yield stress" ( $\sigma_{yma}$ ) followed by flow (creep)
- BC : softening
- C : " minimum yield stress" ( $\sigma_{ymi}$ )
- CD : hardening
- different strain rate dependency

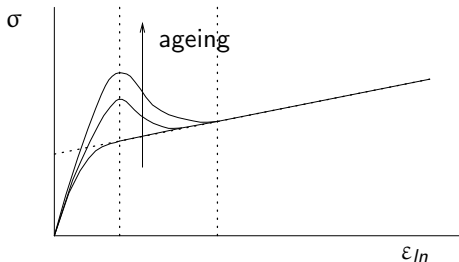
# Strain rate dependency : PC



# Strain rate dependency : PMMA



# Aging

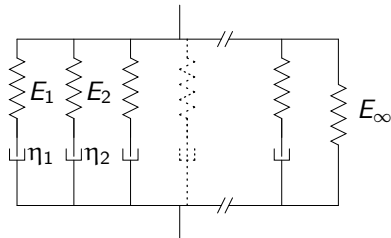


- rejuvenation  $\rightarrow$  no softening
- aging  $\rightarrow$  softening

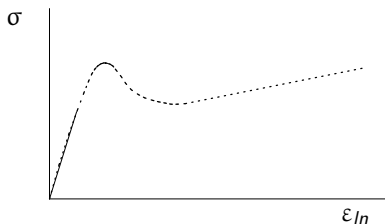
# Nonlinear viscoelastic model

- Linear viscoelastic behavior
- Non-linear viscoelastic behavior
- Softening
- Hardening

# Linear viscoelastic behavior



$$\sigma(t) = \int_{\xi=-\infty}^t E(t-\xi) \dot{\epsilon}(\xi) d\xi \quad ; \quad E(x) = E_{\infty} + \sum_{i=1}^N E_i e^{-\frac{x}{\tau_i}} \quad ; \quad \tau_i = \frac{\eta_i}{E_i}$$

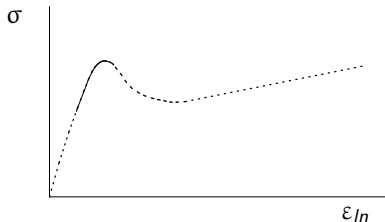


# Nonlinear viscoelastic behavior

$$\sigma(t) = \int_{\xi=-\infty}^t E(\psi - \psi') \dot{\varepsilon}(\xi) d\xi$$

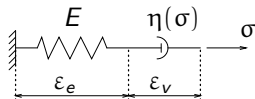
$$\psi = \int_{\zeta=-\infty}^t \frac{d\zeta}{a_{\sigma}\{\sigma(\zeta)\}} \quad ; \quad \psi' = \int_{\zeta=-\infty}^{\xi} \frac{d\zeta}{a_{\sigma}\{\sigma(\zeta)\}}$$

$$E(x) = E_{\infty} + \sum_{i=1}^N E_i e^{-\frac{x}{\tau_i(\sigma)}}$$

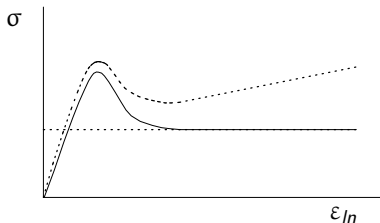




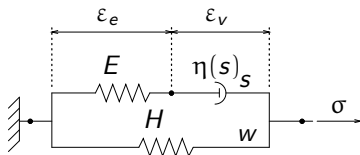
# Softening = tertiary creep



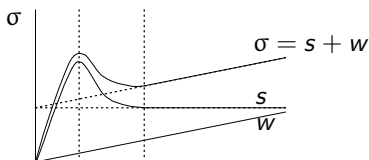
- $\dot{\epsilon} = \dot{\epsilon}_e + \dot{\epsilon}_v$
- $\sigma = E\epsilon_e \rightarrow \dot{\epsilon}_e = \frac{1}{E} \dot{\sigma}$
- $\dot{\epsilon}_v = \frac{1}{\eta(\bar{s}, T, D)} \sigma$  ;  $\bar{s} = |s|$
- $\dot{D} = \left(1 - \frac{D}{D_\infty}\right) h \dot{\epsilon}_v$  ;  $\dot{\epsilon}_v = |\dot{\epsilon}_v|$



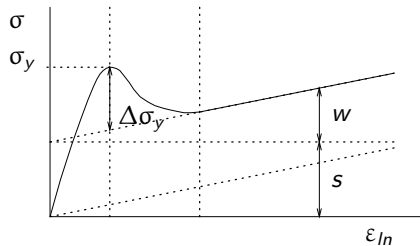
# Hardening



- $\dot{\epsilon} = \dot{\epsilon}_e + \dot{\epsilon}_v$
- $\sigma = s + w = E\epsilon_e + H\epsilon$
- $\dot{\epsilon}_v = \frac{1}{\eta(\bar{s}, T, D)} s \quad ; \quad \bar{s} = |s|$
- $\dot{D} = \left(1 - \frac{D}{D_\infty}\right) h \dot{\bar{\epsilon}}_v \quad ; \quad \dot{\bar{\epsilon}}_v = |\dot{\epsilon}_v|$



# Aging and hardening (newest model)



- $\dot{\varepsilon} = \dot{\varepsilon}_e + \dot{\varepsilon}_v$
- $\sigma = s + \Delta\sigma_y + w = E\varepsilon_e + \Delta\sigma_y + H\varepsilon$
- $\dot{\varepsilon}_v = \frac{1}{\eta(\bar{s}, T, S)} s \quad ; \quad \bar{s} = |s|$

# Aging and hardening (newest model)

- $S(t, \bar{\epsilon}_v) = S_a(t) R_Y(\bar{\epsilon}_v)$
- $R_Y(\bar{\epsilon}_v) = [\{1 + (r_0 e^{\bar{\epsilon}_v})^{r_1}\} / \{1 + r_0^{r_1}\}]^{\frac{r_2-1}{r_1}} \quad ; \quad 0 < R < 1$
- $S_\alpha(t) = S_a(t_{eff}) = c_0 + c_1 \ln \left[ \frac{t_{eff} + t_a}{t_0} \right]$
- $t_{eff}(T, \bar{s}) = \int_{\xi=0}^t \frac{d\xi}{\alpha_T(T(\xi)) \alpha_\sigma(\bar{s}(\xi))}$
- $t_a = \exp \left( \frac{S_\alpha(0) - c_0}{c_1} \right)$
- $\Delta \sigma_y = \sigma_y(t) - \sigma_{y0} = \frac{c}{c_1} \{S_\alpha(t) - c_0\}$

## Eyring viscosity

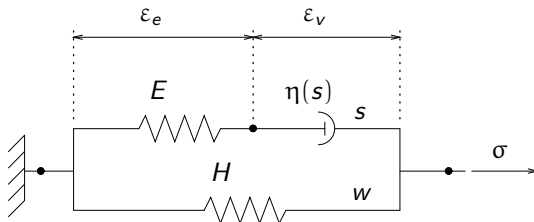
$$\eta = A_0 \frac{\bar{s}}{\sqrt{3} \sinh(\bar{s}/(\sqrt{3}\tau_0))} \exp \left[ \frac{\Delta H}{RT} + \frac{\mu p}{\tau_0} - D \right]$$

$$\bar{s} = |s| \quad ; \quad p = -\frac{1}{3}s \quad ; \quad \tau_0 = \frac{RT}{V}$$

## Bodner-Partom viscosity

$$\eta = \frac{\bar{s}}{\sqrt{12\Gamma_0}} \exp \left[ \frac{1}{2} \left( \frac{Z}{\bar{s}} \right)^{2n} \right]$$
$$Z = Z_1 + (Z_0 - Z_1) \exp [-m\bar{\epsilon}_p]$$

# Nonlinear viscoelastic model older model



- $\dot{\epsilon} = \dot{\epsilon}_e + \dot{\epsilon}_v$
- $\sigma = s + w = E\epsilon_e + H\epsilon$
- $\dot{\epsilon}_v = \frac{1}{\eta(\bar{s}, T, D)} s$  ;  $\bar{s} = |s|$
- $\dot{D} = \left(1 - \frac{D}{D_\infty}\right) h \dot{\epsilon}_v$  ;  $\dot{\epsilon}_v = |\dot{\epsilon}_v|$

# Nonlinear viscoelastic model    older model

constitutive equations

$$\left. \begin{aligned} \dot{\varepsilon}_e &= \dot{\varepsilon} - \dot{\varepsilon}_v = \dot{\varepsilon} - \frac{1}{\eta(\bar{s}, T, D)} s = \dot{\varepsilon} - \frac{E}{\eta(\bar{s}, T, D)} \varepsilon_e \\ \sigma &= s + w = E\varepsilon_e + H\varepsilon \\ \dot{D} &= \left(1 - \frac{D}{D_\infty}\right) h\dot{\varepsilon}_v \end{aligned} \right\}$$



# NUMERICAL IMPLEMENTATION

[back to index](#)

## Stress update

$$\left\{ \begin{array}{l} \dot{\varepsilon}_e = \dot{\varepsilon} - E\zeta(\bar{s}, T, D)\varepsilon_e \\ \dot{D} = \left(1 - \frac{D}{D_\infty}\right) h\dot{\bar{\varepsilon}}_v \end{array} \right.$$
$$\left\{ \begin{array}{l} \Delta\varepsilon_e = \Delta\varepsilon - \Delta t E\zeta(\bar{s}, T, D)\varepsilon_e \\ \Delta D = \left(1 - \frac{D}{D_\infty}\right) h\Delta\bar{\varepsilon}_v \end{array} \right.$$

# Implicit stress update

$$\left\{ \begin{array}{l} \varepsilon_e - \varepsilon_{en} = \varepsilon - \varepsilon_n - \Delta t E \zeta(\bar{s}, T, D) \varepsilon_e \\ D - D_n = \left(1 - \frac{D}{D_\infty}\right) h \Delta \bar{\varepsilon}_v \end{array} \right.$$

$$\left\{ \begin{array}{l} \varepsilon_e^* + \delta \varepsilon_e - \varepsilon_{en} = \varepsilon - \varepsilon_n - \Delta t E \zeta(\bar{s}, T, D^* + \delta D) (\varepsilon_e^* + \delta \varepsilon_e) \\ D^* + \delta D - D_n = \left(1 - \frac{D^* + \delta D}{D_\infty}\right) h \Delta \bar{\varepsilon}_v \end{array} \right.$$

$$\left\{ \begin{array}{l} \delta \varepsilon_e + \Delta t E \zeta(\bar{s}, T, D^*) \delta \varepsilon_e + \Delta t E \frac{\partial \zeta}{\partial D} \delta D \varepsilon_e^* \\ \quad = -\varepsilon_e^* + \varepsilon_{en} + \varepsilon - \varepsilon_n - \Delta t E \zeta(\bar{s}, T, D^*) \varepsilon_e^* \\ \left[1 + \frac{h \Delta \bar{\varepsilon}_v}{D_\infty}\right] \delta D \\ \quad = -D^* + D_n + \left(1 - \frac{D^*}{D_\infty}\right) h \Delta \bar{\varepsilon}_v \end{array} \right.$$

## Explicit stress update

$$\left\{ \begin{array}{l} \varepsilon_e - \varepsilon_{en} = \varepsilon - \varepsilon_n - \Delta t E \zeta(\bar{s}_n, T, D_n) \varepsilon_{en} \\ D - D_n = \left(1 - \frac{D_n}{D_\infty}\right) h \Delta \bar{\varepsilon}_v \end{array} \right.$$
$$\left\{ \begin{array}{l} \varepsilon_e = \varepsilon - \varepsilon_n + \{1 - \Delta t E \zeta(\bar{s}_n, T, D_n)\} \varepsilon_{en} \\ D = D_n + \left(1 - \frac{D_n}{D_\infty}\right) h \Delta \bar{\varepsilon}_v \end{array} \right.$$

## Stiffness : implicit

$$\begin{cases}
 \sigma = s + w = E \varepsilon_e + H \varepsilon \\
 \varepsilon_e - \varepsilon_{en} = \varepsilon - \varepsilon_n - \Delta t E \zeta(\bar{s}, T, D) \varepsilon_e \\
 D - D_n = \left(1 - \frac{D}{D_\infty}\right) h \Delta \bar{\varepsilon}_v
 \end{cases}$$

$$\begin{cases}
 \delta \sigma = E \delta \varepsilon_e + H \delta \varepsilon \\
 \delta \varepsilon_e = \delta \varepsilon - \Delta t E \frac{\partial \zeta}{\partial D} \delta D \varepsilon_e - \Delta t E \zeta(\bar{s}, T, D) \delta \varepsilon_e \\
 \delta D = -\frac{\delta D}{D_\infty} h \Delta \bar{\varepsilon}_v \quad \rightarrow \quad \delta D = 0
 \end{cases}$$

$$\begin{cases}
 \delta \sigma = E \delta \varepsilon_e + H \delta \varepsilon \\
 \delta \varepsilon_e = \delta \varepsilon - \Delta t E \zeta(\bar{s}, T, D) \delta \varepsilon_e \\
 \delta D = 0
 \end{cases}$$

$$C_\varepsilon = \frac{E + H\{1 + \Delta t E \zeta(\bar{s}, T, D)\}}{1 + \Delta t E \zeta(\bar{s}, T, D)}$$

## Stiffness : explicit

$$\left\{ \begin{array}{l} \sigma = s + w = E\varepsilon_e + H\varepsilon \\ \varepsilon_e - \varepsilon_{en} = \varepsilon - \varepsilon_n - \Delta t E \zeta(\bar{s}_n, T, D_n) \varepsilon_e \\ D = D_n + \left(1 - \frac{D_n}{D_\infty}\right) h \Delta \bar{\varepsilon}_v \end{array} \right.$$
$$\left\{ \begin{array}{l} \delta\sigma = E\delta\varepsilon_e + H\delta\varepsilon \\ \delta\varepsilon_e = \delta\varepsilon - \Delta t E \zeta(\bar{s}_n, T, D_n) \delta\varepsilon_e \\ \delta D = 0 \end{array} \right.$$

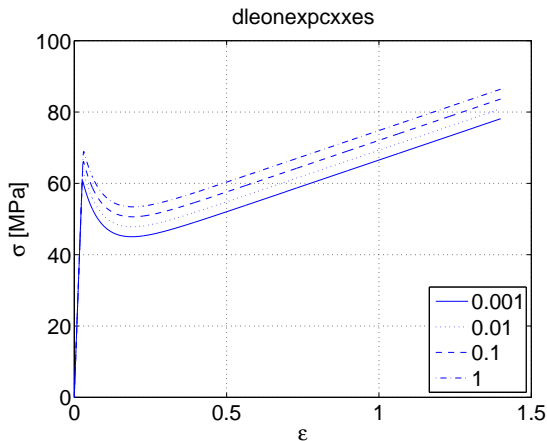
$$\delta\sigma = \frac{E}{1 + \Delta t E \zeta(\bar{s}_n, T, D_n)} \delta\varepsilon + H\delta\varepsilon$$

$$C_\varepsilon = \frac{E + H\{1 + \Delta t E \zeta(\bar{s}_n, T, D_n)\}}{1 + \Delta t E \zeta(\bar{s}_n, T, D_n)}$$

# Examples

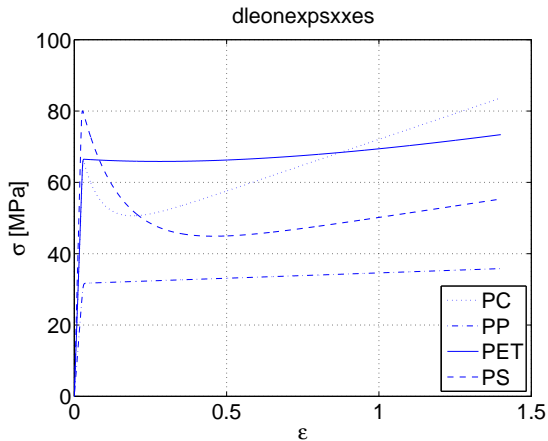
	PET	PC	PS	PP	
$E$	2400	2305	3300	1092	MPa
$\nu$	0.35	0.37	0.37	0.4	-
$H$	15	29	13	3	MPa
$h$	13	270	100	0	-
$D_{\infty}$	11	19	14	-	-
$A_0$	3.8568E-27	9.7573E-27	4.2619E-34	2.0319E-29	s
$\Delta H$	2.617E+05	2.9E+05	2.6E+5	2.2E+5	J/mol
$\mu$	0.0625	0.06984	0.294	0.23	-
$\tau_0$	0.927	0.72	2.1	1.0	MPa

# Tensile test at various strain rates





# Tensile test for various polymers



# VECTORS AND TENSORS

[back to index](#)

# Vectors

$$\vec{a} = \|a\|\vec{e} \quad ; \quad \|\vec{e}\| = 1$$

$$\alpha\vec{a} = \vec{b}$$

$$\vec{a} + \vec{b} = \vec{c}$$

$$\vec{a} \cdot \vec{b} = \|\vec{a}\|\|\vec{b}\| \cos(\phi)$$

$$\vec{c} = \vec{a} * \vec{b} = \left\{ \|\vec{a}\|\|\vec{b}\| \right\} \sin(\phi) \vec{n} \quad ; \quad \|\vec{n}\| = 1$$

$$\vec{a} * \vec{b} \cdot \vec{c} = \left\{ \|\vec{a}\|\|\vec{b}\| \sin(\phi) \right\} \|\vec{c}\| \cos(\psi)$$

$$\vec{a}\vec{b} = \text{dyad} \quad ; \quad \vec{q} = \vec{a}\vec{b} \cdot \vec{p} = \vec{p} \cdot (\vec{a}\vec{b})^c$$

$$\{\vec{e}_1, \vec{e}_2, \vec{e}_3\} \quad ; \quad \vec{e}_i \cdot \vec{e}_{j \neq i} = 0 \quad ; \quad \vec{e}_i \cdot \vec{e}_i = 1$$

$$\vec{a} = \underset{\sim}{a}^T \underset{\sim}{e} = \underset{\sim}{e}^T \underset{\sim}{a}$$

## Second-order tensors

$$\mathbf{A} = \sum_i \alpha_i \vec{a}_i \vec{b}_i \quad ; \quad \mathbf{A} \cdot \vec{p} = \vec{q} \quad ; \quad \mathbf{A} = \vec{e}^T \underline{A} \vec{e}$$

$$\mathbf{I} \cdot \vec{a} = \vec{a} \quad \forall \quad \vec{a} \quad \rightarrow \quad \mathbf{I} = \vec{e}^T \underline{I} \vec{e}$$

$$\mathbf{A}^c = \sum_i \alpha_i \vec{b}_i \vec{a}_i \quad ; \quad \mathbf{A} \cdot \vec{p} = \vec{p} \cdot \mathbf{A}^c$$

$$\alpha \mathbf{A} = \mathbf{B} \quad ; \quad \mathbf{A} + \mathbf{B} = \mathbf{C} \quad ; \quad \mathbf{B} \cdot \mathbf{A} = \mathbf{C}$$

$$\mathbf{A} : \mathbf{B} = \mathbf{A}^c : \mathbf{B}^c = \text{scalar}$$

$$J_1(\mathbf{A}) = \text{tr}(\mathbf{A})$$

$$J_2(\mathbf{A}) = \frac{1}{2} \{ \text{tr}^2(\mathbf{A}) - \text{tr}(\mathbf{A} \cdot \mathbf{A}) \}$$

$$J_3(\mathbf{A}) = \det(\mathbf{A}) \quad ; \quad \det(\mathbf{A}) = 0 \rightarrow \mathbf{A} = \text{singular}$$

$$\mathbf{A}^{-1} \cdot \mathbf{A} = \mathbf{I} \quad ; \quad \mathbf{A} = \text{regular}$$

$$\mathbf{A}^c = \mathbf{A} \quad ; \quad \mathbf{A}^c = -\mathbf{A}$$

$$\vec{a} \cdot \mathbf{A} \cdot \vec{a} > 0 \quad \forall \quad \vec{a} \neq \vec{0}$$

$$(\mathbf{A} \cdot \vec{a}) \cdot (\mathbf{A} \cdot \vec{b}) = \vec{a} \cdot \vec{b} \quad \forall \quad \vec{a}, \vec{b}$$

$$(\mathbf{A} \cdot \vec{a}) * (\mathbf{A} \cdot \vec{b}) = \mathbf{A}^a \cdot (\vec{a} * \vec{b}) \quad \forall \quad \vec{a}, \vec{b}$$

## Fourth-order tensors

$${}^4\mathbf{A} = \sum_i \alpha_i \vec{a}_i \vec{b}_i \vec{c}_i \vec{d}_i \quad ; \quad {}^4\mathbf{A} : \mathbf{B} = \mathbf{C}$$

$${}^4\mathbf{I} : \mathbf{A} = \mathbf{A} \quad \forall \quad \mathbf{A}$$

$${}^4\mathbf{A} \cdot \mathbf{B} = {}^4\mathbf{C}$$

## COLUMN AND MATRIX NOTATION

[back to index](#)

# Matrix/column notation for second-order tensor

$3 \times 3$  matrix of a second-order tensor

$$\mathbf{A} = \vec{e}_i A_{ij} \vec{e}_j \quad \rightarrow \quad \underline{A} = \begin{bmatrix} A_{11} & A_{12} & A_{13} \\ A_{21} & A_{22} & A_{23} \\ A_{31} & A_{32} & A_{33} \end{bmatrix}$$

column notation

$$\underline{A}^T = \begin{bmatrix} A_{11} & A_{21} & A_{31} & A_{12} & A_{22} & A_{32} & A_{13} & A_{23} & A_{33} \end{bmatrix}$$

$$\underline{A}_t^T = \begin{bmatrix} A_{11} & A_{21} & A_{31} & A_{12} & A_{22} & A_{32} & A_{13} & A_{23} & A_{33} \end{bmatrix}$$

conjugate tensor

$$\mathbf{A}^c \quad \rightarrow \quad A_{ji} \quad \rightarrow \quad \underline{A}^T = \begin{bmatrix} A_{11} & A_{21} & A_{31} \\ A_{12} & A_{22} & A_{32} \\ A_{13} & A_{23} & A_{33} \end{bmatrix} \quad \rightarrow \quad \underline{A}_t$$

## Column notation for $\mathbf{A} : \mathbf{B}$

$$\mathbf{C} = \mathbf{A} : \mathbf{B}$$

$$= \vec{e}_i A_{ij} \vec{e}_j : \vec{e}_k B_{kl} \vec{e}_l = A_{ij} \delta_{jk} \delta_{il} B_{kl} = A_{ij} B_{ji}$$

$$= A_{11} B_{11} + A_{12} B_{21} + A_{13} B_{31} +$$

$$A_{21} B_{12} + A_{22} B_{22} + A_{23} B_{32} +$$

$$A_{31} B_{13} + A_{32} B_{23} + A_{33} B_{33}$$

$$= \begin{bmatrix} A_{11} & A_{22} & A_{33} & A_{21} & A_{12} & A_{32} & A_{23} & A_{13} & A_{31} \end{bmatrix}$$

$$\begin{bmatrix} B_{11} & B_{22} & B_{33} & B_{12} & B_{21} & B_{23} & B_{32} & B_{31} & B_{13} \end{bmatrix}^T$$

$$= \underset{\approx_t}{A}^T \underset{\approx}{B} = \underset{\approx}{A}^T \underset{\approx_t}{B}$$

idem

$$\mathbf{C} = \mathbf{A} : \mathbf{B}^c \quad \rightarrow \quad \mathbf{C} = \underset{\approx_t}{A}^T \underset{\approx_t}{B} = \underset{\approx}{A}^T \underset{\approx}{B}$$

$$\mathbf{C} = \mathbf{A}^c : \mathbf{B} \quad \rightarrow \quad \mathbf{C} = \underset{\approx}{A}^T \underset{\approx}{B} = \underset{\approx_t}{A}^T \underset{\approx_t}{B}$$

$$\mathbf{C} = \mathbf{A}^c : \mathbf{B}^c \quad \rightarrow \quad \mathbf{C} = \underset{\approx_t}{A}^T \underset{\approx}{B} = \underset{\approx}{A}^T \underset{\approx_t}{B}$$



# Matrix/column notation $\mathbf{C} = \mathbf{A} \cdot \mathbf{B}$

$$\mathbf{C} = \mathbf{A} \cdot \mathbf{B} = \vec{e}_i A_{ik} \vec{e}_k \cdot \vec{e}_l B_{lj} \vec{e}_j = \vec{e}_i A_{ik} \delta_{kl} B_{lj} \vec{e}_j = \vec{e}_i A_{ik} B_{kj} \vec{e}_j \rightarrow$$

$$\underline{\mathbf{C}} = \begin{bmatrix} A_{11}B_{11} + A_{12}B_{21} + A_{13}B_{31} & A_{11}B_{12} + A_{12}B_{22} + A_{13}B_{32} & A_{11}B_{13} + A_{12}B_{23} + A_{13}B_{33} \\ A_{21}B_{11} + A_{22}B_{21} + A_{23}B_{31} & A_{21}B_{12} + A_{22}B_{22} + A_{23}B_{32} & A_{21}B_{13} + A_{22}B_{23} + A_{23}B_{33} \\ A_{31}B_{11} + A_{32}B_{21} + A_{33}B_{31} & A_{31}B_{12} + A_{32}B_{22} + A_{33}B_{32} & A_{31}B_{13} + A_{32}B_{23} + A_{33}B_{33} \end{bmatrix}$$

$$\underset{\approx}{\mathbf{C}} = \begin{bmatrix} C_{11} \\ C_{22} \\ C_{33} \\ C_{12} \\ C_{21} \\ C_{23} \\ C_{32} \\ C_{31} \\ C_{13} \end{bmatrix} = \begin{bmatrix} A_{11}B_{11} + A_{12}B_{21} + A_{13}B_{31} \\ A_{21}B_{12} + A_{22}B_{22} + A_{23}B_{32} \\ A_{31}B_{13} + A_{32}B_{23} + A_{33}B_{33} \\ A_{11}B_{12} + A_{12}B_{22} + A_{13}B_{32} \\ A_{21}B_{11} + A_{22}B_{21} + A_{23}B_{31} \\ A_{21}B_{13} + A_{22}B_{23} + A_{23}B_{33} \\ A_{31}B_{12} + A_{32}B_{22} + A_{33}B_{32} \\ A_{31}B_{11} + A_{32}B_{21} + A_{33}B_{31} \\ A_{11}B_{13} + A_{12}B_{23} + A_{13}B_{33} \end{bmatrix}$$

# Matrix/column notation $\mathbf{C} = \mathbf{A} \cdot \mathbf{B}$

$$\underset{\approx}{\mathbf{C}} = \begin{bmatrix} A_{11} & 0 & 0 & 0 & A_{12} & 0 & 0 & A_{13} & 0 \\ 0 & A_{22} & 0 & A_{21} & 0 & 0 & A_{23} & 0 & 0 \\ 0 & 0 & A_{33} & 0 & 0 & A_{32} & 0 & 0 & A_{31} \\ 0 & A_{12} & 0 & A_{11} & 0 & 0 & A_{13} & 0 & 0 \\ A_{21} & 0 & 0 & 0 & A_{22} & 0 & 0 & A_{23} & 0 \\ 0 & 0 & A_{23} & 0 & 0 & A_{22} & 0 & 0 & A_{21} \\ 0 & A_{32} & 0 & A_{31} & 0 & 0 & A_{33} & 0 & 0 \\ A_{31} & 0 & 0 & 0 & A_{32} & 0 & 0 & A_{33} & 0 \\ 0 & 0 & A_{13} & 0 & 0 & A_{12} & 0 & 0 & A_{11} \end{bmatrix} \begin{bmatrix} B_{11} \\ B_{22} \\ B_{33} \\ B_{12} \\ B_{21} \\ B_{23} \\ B_{32} \\ B_{31} \\ B_{13} \end{bmatrix} = \underline{\underline{\underset{\approx}{\mathbf{A}}}} \underset{\approx}{\mathbf{B}}$$

$$\begin{aligned} \mathbf{C} &= \mathbf{A} \cdot \mathbf{B} & \rightarrow & \underset{\approx}{\mathbf{C}} = \underline{\underline{\underset{\approx}{\mathbf{A}}}} \underset{\approx}{\mathbf{B}} = \underline{\underline{\underset{\approx}{\mathbf{A}}}}_{\underset{\approx}{c}} \underset{\approx}{B}_t \\ & & & \underset{\approx}{C}_t = \underline{\underline{\underset{\approx}{\mathbf{A}}}}_{\underset{\approx}{r}} \underset{\approx}{B} = \underline{\underline{\underset{\approx}{\mathbf{A}}}}_{rc} \underset{\approx}{B}_t \\ \mathbf{C} &= \mathbf{A} \cdot \mathbf{B}^c & \rightarrow & \underset{\approx}{\mathbf{C}} = \underline{\underline{\underset{\approx}{\mathbf{A}}}} \underset{\approx}{B}_t = \underline{\underline{\underset{\approx}{\mathbf{A}}}}_{\underset{\approx}{c}} \underset{\approx}{B} \\ \mathbf{C} &= \mathbf{A}^c \cdot \mathbf{B} & \rightarrow & \underset{\approx}{\mathbf{C}} = \underline{\underline{\underset{\approx}{\mathbf{A}}}}_t \underset{\approx}{B} = \underline{\underline{\underset{\approx}{\mathbf{A}}}}_{tc} \underset{\approx}{B}_t \\ \mathbf{C} &= \mathbf{A}^c \cdot \mathbf{B}^c & \rightarrow & \underset{\approx}{\mathbf{C}} = \underline{\underline{\underset{\approx}{\mathbf{A}}}}_t \underset{\approx}{B}_t = \underline{\underline{\underset{\approx}{\mathbf{A}}}}_{tc} \underset{\approx}{B} \end{aligned}$$

# Matrix notation of fourth-order tensor

$${}^4\mathbf{A} = \vec{e}_i \vec{e}_j A_{ijkl} \vec{e}_k \vec{e}_l \rightarrow$$

$$\underline{\underline{A}} = \begin{bmatrix} A_{1111} & A_{1122} & A_{1133} & A_{1112} & A_{1121} & A_{1123} & A_{1132} & A_{1131} & A_{1113} \\ A_{2211} & A_{2222} & A_{2233} & A_{2212} & A_{2221} & A_{2223} & A_{2232} & A_{2231} & A_{2213} \\ A_{3311} & A_{3322} & A_{3333} & A_{3312} & A_{3321} & A_{3323} & A_{3332} & A_{3331} & A_{3313} \\ A_{1211} & A_{1222} & A_{1233} & A_{1212} & A_{1221} & A_{1223} & A_{1232} & A_{1231} & A_{1213} \\ A_{2111} & A_{2122} & A_{2133} & A_{2112} & A_{2121} & A_{2123} & A_{2132} & A_{2131} & A_{2113} \\ A_{2311} & A_{2322} & A_{2333} & A_{2312} & A_{2321} & A_{2323} & A_{2332} & A_{2331} & A_{2313} \\ A_{3211} & A_{3222} & A_{3233} & A_{3212} & A_{3221} & A_{3223} & A_{3232} & A_{3231} & A_{3213} \\ A_{3111} & A_{3122} & A_{3133} & A_{3112} & A_{3121} & A_{3123} & A_{3132} & A_{3131} & A_{3113} \\ A_{1311} & A_{1322} & A_{1333} & A_{1312} & A_{1321} & A_{1323} & A_{1332} & A_{1331} & A_{1313} \end{bmatrix}$$

$${}^4\mathbf{A}^c \rightarrow \underline{\underline{A}}^T \quad ; \quad {}^4\mathbf{A}^{rc} \rightarrow \underline{\underline{A}}_c \quad ; \quad {}^4\mathbf{A}^{lc} \rightarrow \underline{\underline{A}}_r$$

# Matrix/column notation $\mathbf{C} = {}^4\mathbf{A} : \mathbf{B}$

$$\mathbf{C} = {}^4\mathbf{A} : \mathbf{B} \rightarrow$$

$$\begin{aligned}\vec{e}_i C_{ij} \vec{e}_j &= \vec{e}_i \vec{e}_j A_{ijmn} \vec{e}_m \vec{e}_n : \vec{e}_p B_{pq} \vec{e}_q \\ &= \vec{e}_i \vec{e}_j A_{ijmn} \delta_{np} \delta_{mq} B_{pq} = \vec{e}_i \vec{e}_j A_{ijmn} B_{nm} \rightarrow\end{aligned}$$

$$\underline{\underline{C}}_{\approx} = \underline{\underline{A}}_{\approx c} \underline{\underline{B}}_{\approx} = \underline{\underline{A}}_{\approx} \underline{\underline{B}}_{\approx t}$$

$$\mathbf{C} = \mathbf{B} : {}^4\mathbf{A} \rightarrow$$

$$\begin{aligned}\vec{e}_i C_{ij} \vec{e}_j &= \vec{e}_p B_{pq} \vec{e}_q : \vec{e}_m \vec{e}_n A_{mnij} \vec{e}_i \vec{e}_j \\ &= B_{pq} \delta_{qm} \delta_{pn} A_{mnij} \vec{e}_i \vec{e}_j = B_{nm} A_{mnij} \vec{e}_i \vec{e}_j \rightarrow\end{aligned}$$

$$\underline{\underline{C}}_{\approx}^T = \underline{\underline{B}}_{\approx}^T \underline{\underline{A}}_{\approx r} = \underline{\underline{B}}_{\approx t}^T \underline{\underline{A}}_{\approx}$$

# Matrix notation ${}^4\mathbf{C} = {}^4\mathbf{A} \cdot \mathbf{B}$

$$\begin{aligned}
 {}^4\mathbf{C} &= {}^4\mathbf{A} \cdot \mathbf{B} = \vec{e}_i \vec{e}_j A_{ijkl} \vec{e}_k \vec{e}_l \cdot \vec{e}_p B_{pq} \vec{e}_q \\
 &= \vec{e}_i \vec{e}_j A_{ijkl} \vec{e}_k \delta_{lp} B_{pq} \vec{e}_q = \vec{e}_i \vec{e}_j A_{ijkl} B_{lq} \vec{e}_k \vec{e}_q \\
 &= \vec{e}_i \vec{e}_j A_{ijkp} B_{pl} \vec{e}_k \vec{e}_l \rightarrow
 \end{aligned}$$

$$\begin{aligned}
 \underline{\underline{\mathbf{C}}} &= \begin{bmatrix} A_{111p} B_{p1} & A_{112p} B_{p2} & A_{113p} B_{p3} & A_{111p} B_{p2} & A_{112p} B_{p1} \\ A_{221p} B_{p1} & A_{222p} B_{p2} & A_{223p} B_{p3} & A_{221p} B_{p2} & A_{222p} B_{p1} \\ A_{331p} B_{p1} & A_{332p} B_{p2} & A_{333p} B_{p3} & A_{331p} B_{p2} & A_{332p} B_{p1} \\ A_{121p} B_{p1} & A_{122p} B_{p2} & A_{123p} B_{p3} & A_{121p} B_{p2} & A_{122p} B_{p1} \\ A_{211p} B_{p1} & A_{212p} B_{p2} & A_{213p} B_{p3} & A_{211p} B_{p2} & A_{212p} B_{p1} \end{bmatrix} \\
 &= \begin{bmatrix} A_{1111} & A_{1122} & A_{1133} & A_{1112} & A_{1121} \\ A_{2211} & A_{2222} & A_{2233} & A_{2212} & A_{2221} \\ A_{3311} & A_{3322} & A_{3333} & A_{3312} & A_{3321} \\ A_{1211} & A_{1222} & A_{1233} & A_{1212} & A_{1221} \\ A_{2111} & A_{2122} & A_{2133} & A_{2112} & A_{2121} \end{bmatrix} \begin{bmatrix} B_{11} & 0 & 0 & B_{12} & 0 \\ 0 & B_{22} & 0 & 0 & B_{21} \\ 0 & 0 & B_{33} & 0 & 0 \\ B_{21} & 0 & 0 & B_{22} & 0 \\ 0 & B_{12} & 0 & 0 & B_{11} \end{bmatrix} \\
 &= \underline{\underline{\mathbf{A}}} \underline{\underline{\mathbf{B}}} = \underline{\underline{\mathbf{A}}} \underline{\underline{\mathbf{B}}}_c \rightarrow \underline{\underline{\mathbf{C}}}_r = \underline{\underline{\mathbf{A}}}_r \underline{\underline{\mathbf{B}}}_{cr} = \underline{\underline{\mathbf{A}}}_{cr} \underline{\underline{\mathbf{B}}}_c
 \end{aligned}$$

# Matrix notation ${}^4\mathbf{C} = \mathbf{B} \cdot {}^4\mathbf{A}$

$$\begin{aligned}
 {}^4\mathbf{C} = \mathbf{B} \cdot {}^4\mathbf{A} &= \vec{e}_i B_{ij} \vec{e}_j \cdot \vec{e}_p \vec{e}_q A_{pqrs} \vec{e}_r \vec{e}_s \\
 &= \vec{e}_i B_{ij} \delta_{jp} \vec{e}_q A_{pqrs} \vec{e}_r \vec{e}_s = \vec{e}_i \vec{e}_q B_{ij} A_{jqrs} \vec{e}_r \vec{e}_s \\
 &= \vec{e}_i \vec{e}_j B_{ip} A_{pjkl} \vec{e}_k \vec{e}_l \rightarrow
 \end{aligned}$$

$$\begin{aligned}
 \underline{\underline{\mathbf{C}}} &= \begin{bmatrix} B_{1p} A_{p111} & B_{1p} A_{p122} & B_{1p} A_{p133} & B_{1p} A_{p112} & B_{1p} A_{p121} \\ B_{2p} A_{p211} & B_{2p} A_{p222} & B_{2p} A_{p233} & B_{2p} A_{p212} & B_{2p} A_{p221} \\ B_{3p} A_{p311} & B_{3p} A_{p322} & B_{3p} A_{p333} & B_{3p} A_{p312} & B_{3p} A_{p321} \\ B_{1p} A_{p211} & B_{1p} A_{p222} & B_{1p} A_{p233} & B_{1p} A_{p212} & B_{1p} A_{p221} \\ B_{2p} A_{p111} & B_{2p} A_{p122} & B_{2p} A_{p133} & B_{2p} A_{p112} & B_{2p} A_{p121} \end{bmatrix} \\
 &= \begin{bmatrix} B_{11} & 0 & 0 & 0 & B_{12} \\ 0 & B_{22} & 0 & B_{21} & 0 \\ 0 & 0 & B_{33} & 0 & 0 \\ 0 & B_{12} & 0 & B_{11} & 0 \\ B_{21} & 0 & 0 & 0 & B_{22} \end{bmatrix} \begin{bmatrix} A_{1111} & A_{1122} & A_{1133} & A_{1112} & A_{1121} \\ A_{2211} & A_{2222} & A_{2233} & A_{2212} & A_{2221} \\ A_{3311} & A_{3322} & A_{3333} & A_{3312} & A_{3321} \\ A_{1211} & A_{1222} & A_{1233} & A_{1212} & A_{1221} \\ A_{2111} & A_{2122} & A_{2133} & A_{2112} & A_{2121} \end{bmatrix} \\
 &= \underline{\underline{\mathbf{B}}} \underline{\underline{\mathbf{A}}} = \underline{\underline{\mathbf{B}}}_c \underline{\underline{\mathbf{A}}}_r \rightarrow \underline{\underline{\mathbf{C}}}_r = \underline{\underline{\mathbf{B}}}_r \underline{\underline{\mathbf{A}}}_c = \underline{\underline{\mathbf{B}}}_{cr} \underline{\underline{\mathbf{A}}}_{cr}
 \end{aligned}$$

# Matrix notation ${}^4\mathbf{C} = {}^4\mathbf{A} : {}^4\mathbf{B}$

$$\begin{aligned} {}^4\mathbf{C} = {}^4\mathbf{A} : {}^4\mathbf{B} &= \vec{e}_i \vec{e}_j A_{ijkl} \vec{e}_k \vec{e}_l : \vec{e}_p \vec{e}_q B_{pqrs} \vec{e}_r \vec{e}_s \\ &= \vec{e}_i \vec{e}_j A_{ijkl} \delta_{lp} \delta_{kq} B_{pqrs} \vec{e}_r \vec{e}_s = \vec{e}_i \vec{e}_j A_{ijqp} B_{pqrs} \vec{e}_r \vec{e}_s \\ &= \vec{e}_i \vec{e}_j A_{ijqp} B_{pqkl} \vec{e}_k \vec{e}_l \end{aligned}$$

$$\begin{aligned} \underline{\underline{\mathbf{C}}} &= \begin{bmatrix} A_{11qp} B_{pq11} & A_{11qp} B_{pq22} & A_{11qp} B_{pq33} & A_{11qp} B_{pq12} & A_{11qp} B_{pq21} \\ A_{22qp} B_{pq11} & A_{22qp} B_{pq22} & A_{22qp} B_{pq33} & A_{22qp} B_{pq12} & A_{22qp} B_{pq21} \\ A_{33qp} B_{pq11} & A_{33qp} B_{pq22} & A_{33qp} B_{pq33} & A_{33qp} B_{pq12} & A_{33qp} B_{pq21} \\ A_{12qp} B_{pq11} & A_{12qp} B_{pq22} & A_{12qp} B_{pq33} & A_{12qp} B_{pq12} & A_{12qp} B_{pq21} \\ A_{21qp} B_{pq11} & A_{21qp} B_{pq22} & A_{21qp} B_{pq33} & A_{21qp} B_{pq12} & A_{21qp} B_{pq21} \end{bmatrix} \\ &= \begin{bmatrix} A_{1111} & A_{1122} & A_{1133} & A_{1112} & A_{1121} \\ A_{2211} & A_{2222} & A_{2233} & A_{2212} & A_{2221} \\ A_{3311} & A_{3322} & A_{3333} & A_{3312} & A_{3321} \\ A_{1211} & A_{1222} & A_{1233} & A_{1212} & A_{1221} \\ A_{2111} & A_{2122} & A_{2133} & A_{2112} & A_{2121} \end{bmatrix} \begin{bmatrix} B_{1111} & B_{1122} & B_{1133} & B_{1112} & B_{1121} \\ B_{2211} & B_{2222} & B_{2233} & B_{2212} & B_{2221} \\ B_{3311} & B_{3322} & B_{3333} & B_{3312} & B_{3321} \\ B_{2111} & B_{2122} & B_{2133} & B_{2112} & B_{2121} \\ B_{1211} & B_{1222} & B_{1233} & B_{1212} & B_{1221} \end{bmatrix} \\ &= \underline{\underline{\mathbf{A}}} \underline{\underline{\mathbf{B}}}_r = \underline{\underline{\mathbf{A}}}_c \underline{\underline{\mathbf{B}}} \end{aligned}$$

# Matrix notation fourth-order unit tensor

$${}^4\mathbf{I} = \vec{e}_i \vec{e}_j \delta_{il} \delta_{jk} \vec{e}_k \vec{e}_l \rightarrow$$

$$\underline{\underline{I}} =$$

$$\begin{bmatrix} \delta_{11}\delta_{11} & \delta_{12}\delta_{12} & \delta_{13}\delta_{13} & \delta_{12}\delta_{11} & \delta_{11}\delta_{12} & \cdot \\ \delta_{21}\delta_{21} & \delta_{22}\delta_{22} & \delta_{23}\delta_{23} & \delta_{22}\delta_{21} & \delta_{21}\delta_{22} & \cdot \\ \delta_{31}\delta_{31} & \delta_{32}\delta_{32} & \delta_{33}\delta_{33} & \delta_{32}\delta_{31} & \delta_{31}\delta_{32} & \cdot \\ \delta_{11}\delta_{21} & \delta_{12}\delta_{22} & \delta_{13}\delta_{23} & \delta_{12}\delta_{21} & \delta_{11}\delta_{22} & \cdot \\ \delta_{21}\delta_{11} & \delta_{22}\delta_{12} & \delta_{23}\delta_{13} & \delta_{22}\delta_{11} & \delta_{21}\delta_{12} & \cdot \\ \cdot & \cdot & \cdot & \cdot & \cdot & \cdot \end{bmatrix} = \begin{bmatrix} 1 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 1 & 0 & 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 1 & 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 1 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 1 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 & 1 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 1 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 1 \\ 0 & 0 & 0 & 0 & 0 & 0 & 0 & 1 & 0 \end{bmatrix}$$

symmetric fourth-order tensor

$${}^4\mathbf{I}^s = \frac{1}{2} ({}^4\mathbf{I} + {}^4\mathbf{I}^{rc}) \rightarrow \underline{\underline{I}}^s = \frac{1}{2} (\underline{\underline{I}} + \underline{\underline{I}}_c) = \frac{1}{2} \begin{bmatrix} 2 & 0 & 0 & 0 & 0 & \cdot \\ 0 & 2 & 0 & 0 & 0 & \cdot \\ 0 & 0 & 2 & 0 & 0 & \cdot \\ 0 & 0 & 0 & 1 & 1 & \cdot \\ 0 & 0 & 0 & 1 & 1 & \cdot \\ \cdot & \cdot & \cdot & \cdot & \cdot & \cdot \end{bmatrix}$$



# Matrix notation **I**

$$\mathbf{I} = \vec{e}_i \delta_{ij} \vec{e}_j \vec{e}_k \delta_{kl} \vec{e}_l = \vec{e}_i \vec{e}_j \delta_{ij} \delta_{kl} \vec{e}_k \vec{e}_l \rightarrow$$

$$\mathbf{I} = \begin{bmatrix} 1 & 1 & 1 & 0 & 0 & 0 & 0 & 0 & 0 \\ 1 & 1 & 1 & 0 & 0 & 0 & 0 & 0 & 0 \\ 1 & 1 & 1 & 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 \end{bmatrix}$$

$$= \mathbf{I}^T$$

$$\approx \approx$$

# Matrix notation ${}^4\mathbf{B} = {}^4\mathbf{I} \cdot \mathbf{A}$

$$\begin{aligned}
 {}^4\mathbf{B} &= {}^4\mathbf{I} \cdot \mathbf{A} = \vec{e}_i \vec{e}_j \delta_{il} \delta_{jk} \vec{e}_k \vec{e}_l \cdot \vec{e}_p A_{pq} \vec{e}_q \\
 &= \vec{e}_i \vec{e}_j \delta_{il} \delta_{jk} \vec{e}_k \delta_{lp} A_{pq} \vec{e}_q = \vec{e}_i \vec{e}_j \delta_{il} \delta_{jk} A_{lq} \vec{e}_k \vec{e}_q \\
 &= \vec{e}_i \vec{e}_j A_{iq} \delta_{jk} \vec{e}_k \vec{e}_q = \vec{e}_i \vec{e}_j A_{il} \delta_{jk} \vec{e}_k \vec{e}_l \\
 &= \mathbf{A} \cdot {}^4\mathbf{I} \rightarrow
 \end{aligned}$$

$$\begin{aligned}
 \underline{\underline{\mathbf{B}}} &= \begin{bmatrix} A_{11}\delta_{11} & A_{12}\delta_{12} & A_{13}\delta_{13} & A_{12}\delta_{11} & A_{11}\delta_{12} & .. \\ A_{21}\delta_{21} & A_{22}\delta_{22} & A_{23}\delta_{23} & A_{22}\delta_{21} & A_{21}\delta_{22} & .. \\ A_{31}\delta_{31} & A_{32}\delta_{32} & A_{33}\delta_{33} & A_{32}\delta_{31} & A_{31}\delta_{32} & .. \\ A_{11}\delta_{21} & A_{12}\delta_{22} & A_{13}\delta_{23} & A_{12}\delta_{21} & A_{11}\delta_{22} & .. \\ A_{21}\delta_{11} & A_{22}\delta_{12} & A_{23}\delta_{13} & A_{22}\delta_{11} & A_{21}\delta_{12} & .. \\ .. & .. & .. & .. & .. & .. \end{bmatrix} \\
 &= \begin{bmatrix} A_{11} & 0 & 0 & A_{12} & 0 & .. \\ 0 & A_{22} & 0 & 0 & A_{21} & .. \\ 0 & 0 & A_{33} & 0 & 0 & .. \\ 0 & A_{12} & 0 & 0 & A_{11} & .. \\ A_{21} & 0 & 0 & A_{22} & 0 & .. \\ .. & .. & .. & .. & .. & .. \end{bmatrix} \\
 &= \underline{\underline{\mathbf{A}}}_c
 \end{aligned}$$

# Summary and examples

$$\begin{aligned}
 \vec{x} &\rightarrow \underline{x} \\
 \mathbf{A} &\rightarrow \underline{A} \quad ; \quad \underline{\underline{A}} \quad ; \quad \underline{\underline{\underline{A}}} \\
 {}^4\mathbf{A} &\rightarrow \underline{\underline{A}} \\
 {}^4\mathbf{I} &\rightarrow \underline{\underline{I}}
 \end{aligned}$$

$$\underline{x} = \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} \quad ; \quad \underline{A} = \begin{bmatrix} A_{11} & A_{12} & A_{13} \\ A_{21} & A_{22} & A_{23} \\ A_{31} & A_{32} & A_{33} \end{bmatrix}$$

$$\underline{\underline{A}} = \begin{bmatrix} A_{11} \\ A_{22} \\ A_{33} \\ A_{12} \\ A_{21} \\ \dots \end{bmatrix} \quad ; \quad \underline{\underline{\underline{A}}} = \begin{bmatrix} A_{11} & 0 & 0 & 0 & A_{12} & \dots \\ 0 & A_{22} & 0 & A_{21} & 0 & \dots \\ 0 & 0 & A_{33} & 0 & 0 & \dots \\ 0 & A_{12} & 0 & A_{11} & 0 & \dots \\ A_{21} & 0 & 0 & 0 & A_{22} & \dots \\ \dots & \dots & \dots & \dots & \dots & \dots \end{bmatrix}$$

$$\underline{\underline{\underline{A}}} = \begin{bmatrix} A_{1111} & A_{1112} & A_{1133} & A_{1112} & A_{1121} & \dots \\ A_{2211} & A_{2222} & A_{2233} & A_{2212} & A_{2221} & \dots \\ A_{3311} & A_{3322} & A_{3333} & A_{3312} & A_{3321} & \dots \\ A_{1211} & A_{1222} & A_{1233} & A_{1212} & A_{1221} & \dots \\ A_{2111} & A_{2122} & A_{2133} & A_{2112} & A_{2121} & \dots \end{bmatrix}$$

# Manipulations

$$\mathbf{A} \rightarrow \underline{A} : \rightarrow \mathbf{mA}$$

$$\mathbf{A} \rightarrow \underline{\approx} : \rightarrow \mathbf{ccA} = \mathbf{m2cc}(\mathbf{mA}, 9)$$

$$\mathbf{A} \rightarrow \underline{\underline{A}} : \rightarrow \mathbf{mmA} = \mathbf{m2mm}(\mathbf{mA}, 9)$$

$$\mathbf{A}^c \rightarrow \underline{A}^T : \rightarrow \mathbf{mA}t = \mathbf{mA}'$$

$$\mathbf{A}^c \rightarrow \underline{\approx}_t : \rightarrow \mathbf{ccAt} = \mathbf{m2cc}(\mathbf{mA}t, 9)$$

$$\mathbf{A}^c \rightarrow \underline{\underline{A}}_t : \rightarrow \mathbf{mmAt} = \mathbf{m2mm}(\mathbf{mA}')$$

$${}^4\mathbf{A}^{lc} \rightarrow \underline{\underline{A}}_r : \rightarrow \mathbf{mmAr} = \mathbf{mmA}([1\ 2\ 3\ 5\ 4\ 7\ 6\ 9\ 8], :)$$

$${}^4\mathbf{A}^{rc} \rightarrow \underline{\underline{A}}_c : \rightarrow \mathbf{mmAc} = \mathbf{mmA}(:, [1\ 2\ 3\ 5\ 4\ 7\ 6\ 9\ 8])$$

# Gradients

## Cartesian

$$\vec{\nabla} = \vec{e}_x \frac{\partial}{\partial x} + \vec{e}_y \frac{\partial}{\partial y} + \vec{e}_z \frac{\partial}{\partial z} = \begin{bmatrix} \vec{e}_x & \vec{e}_y & \vec{e}_z \end{bmatrix} \begin{bmatrix} \frac{\partial}{\partial x} \\ \frac{\partial}{\partial y} \\ \frac{\partial}{\partial z} \end{bmatrix} = \vec{e}^T \nabla$$

## cylindrical

$$\vec{\nabla} = \vec{e}_r \frac{\partial}{\partial r} + \vec{e}_\theta \frac{1}{r} \frac{\partial}{\partial \theta} + \vec{e}_z \frac{\partial}{\partial z} = \begin{bmatrix} \vec{e}_r & \vec{e}_\theta & \vec{e}_z \end{bmatrix} \begin{bmatrix} \frac{\partial}{\partial r} \\ \frac{1}{r} \frac{\partial}{\partial \theta} \\ \frac{\partial}{\partial z} \end{bmatrix} = \vec{e}^T \nabla$$

## Gradient of a vector in Cartesian coordinates

$$\begin{aligned}
 \vec{\nabla} \vec{a} &= \underline{\mathbf{L}}_a^c = \left( \vec{e}_x \frac{\partial}{\partial x} + \vec{e}_y \frac{\partial}{\partial y} + \vec{e}_z \frac{\partial}{\partial z} \right) (a_x \vec{e}_x + a_y \vec{e}_y + a_z \vec{e}_z) \\
 &= \vec{e}_x a_{x,x} \vec{e}_x + \vec{e}_x a_{y,x} \vec{e}_y + \vec{e}_x a_{z,x} \vec{e}_z + \vec{e}_y a_{x,y} \vec{e}_x + \\
 &\quad \vec{e}_y a_{y,y} \vec{e}_y + \vec{e}_y a_{z,y} \vec{e}_z + \vec{e}_z a_{x,z} \vec{e}_x + \vec{e}_z a_{y,z} \vec{e}_y + \vec{e}_z a_{z,z} \vec{e}_z \\
 \underline{\mathbf{L}}_a &= \begin{bmatrix} a_{x,x} & a_{x,y} & a_{x,z} \\ a_{y,x} & a_{y,y} & a_{y,z} \\ a_{z,x} & a_{z,y} & a_{z,z} \end{bmatrix} \\
 \underline{\mathbf{L}}_a^T &= \begin{bmatrix} a_{x,x} & a_{y,y} & a_{z,z} & a_{x,y} & a_{y,x} & a_{y,z} & a_{z,y} & a_{z,x} & a_{x,z} \end{bmatrix}
 \end{aligned}$$

# Gradient of a vector in cylindrical coordinates

$$\vec{\nabla} \vec{a} = \mathbf{L}_a^c = \left( \vec{e}_r \frac{\partial}{\partial r} + \vec{e}_t \frac{1}{r} \frac{\partial}{\partial \theta} + \vec{e}_z \frac{\partial}{\partial z} \right) (a_r \vec{e}_r + a_t \vec{e}_t + a_z \vec{e}_z)$$

$$= \vec{e}_r a_{r,r} \vec{e}_r + \vec{e}_r a_{t,r} \vec{e}_t + \vec{e}_r a_{z,r} \vec{e}_z +$$

$$\vec{e}_t \frac{1}{r} a_{r,t} \vec{e}_r + \vec{e}_t \frac{1}{r} a_{t,t} \vec{e}_t + \vec{e}_t \frac{1}{r} a_{z,t} \vec{e}_z + \vec{e}_t \frac{1}{r} a_r \vec{e}_t - \vec{e}_t \frac{1}{r} a_t \vec{e}_r$$

$$\vec{e}_z a_{r,z} \vec{e}_r + \vec{e}_z a_{t,z} \vec{e}_t + \vec{e}_z a_{z,z} \vec{e}_z$$

$$\underline{L}_a = \begin{bmatrix} a_{r,r} & \frac{1}{r} a_{r,t} - \frac{1}{r} a_t & a_{r,z} \\ a_{t,r} & \frac{1}{r} a_{t,t} + \frac{1}{r} a_r & a_{t,z} \\ a_{z,r} & \frac{1}{r} a_{z,t} & a_{z,z} \end{bmatrix}$$

$$\underline{L}_a^T = \begin{bmatrix} a_{r,r} & \frac{1}{r} a_{t,t} + \frac{1}{r} a_r & a_{z,z} & \frac{1}{r} a_{r,t} - \frac{1}{r} a_t & a_{t,r} & a_{t,z} & \frac{1}{r} a_{z,t} & a_{z,r} & a_{r,z} \end{bmatrix}$$

# Divergence of tensor in cylindrical coordinates

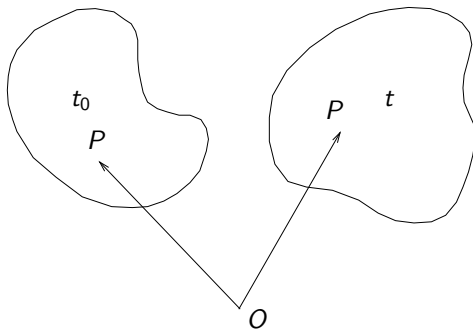
$$\begin{aligned}
 \vec{\nabla} \cdot \mathbf{A} &= \vec{e}_i \cdot \nabla_i (\vec{e}_j A_{jk} \vec{e}_k) \\
 &= \vec{e}_i \cdot (\nabla_i \vec{e}_j) A_{jk} \vec{e}_k + \vec{e}_i \cdot \vec{e}_j (\nabla_i A_{jk}) \vec{e}_k + \vec{e}_i \cdot \vec{e}_j A_{jk} (\nabla_i \vec{e}_k) \\
 &= \vec{e}_i \cdot (\nabla_i \vec{e}_j) A_{jk} \vec{e}_k + \delta_{ij} (\nabla_i A_{jk}) \vec{e}_k + \delta_{ij} A_{jk} (\nabla_i \vec{e}_k) \\
 &\quad \nabla_i \vec{e}_j = \delta_{i2} \delta_{1j} \frac{1}{r} \vec{e}_t - \delta_{i2} \delta_{2j} \frac{1}{r} \vec{e}_r \\
 &= \delta_{1j} \frac{1}{r} A_{jk} \vec{e}_k + (\nabla_j A_{jk}) \vec{e}_k + (\delta_{j2} \delta_{1k} \frac{1}{r} \vec{e}_t - \delta_{j2} \delta_{2k} \frac{1}{r} \vec{e}_r) A_{jk} \\
 &= \frac{1}{r} A_{1k} \vec{e}_k + (\nabla_j A_{jk}) \vec{e}_k + \frac{1}{r} (A_{21} \vec{e}_t - A_{22} \vec{e}_r) \\
 &= \left( \frac{1}{r} A_{11} - \frac{1}{r} A_{22} \right) \vec{e}_1 + \left( \frac{1}{r} A_{12} + \frac{1}{r} A_{21} \right) \vec{e}_2 + \frac{1}{r} A_{13} \vec{e}_3 + \\
 &\quad (\nabla_j A_{jk}) \vec{e}_k \\
 &= g_k \vec{e}_k + \nabla_j A_{jk} \vec{e}_k \\
 &= \underline{\underline{g}}^T \underline{\underline{e}} + (\underline{\underline{\nabla}}^T \underline{\underline{A}}) \underline{\underline{e}} \\
 &= (\underline{\underline{\nabla}}^T \underline{\underline{A}}) \underline{\underline{e}} + \underline{\underline{g}}^T \underline{\underline{e}} \\
 &\quad \text{with} \quad \underline{\underline{g}}^T = \frac{1}{r} \begin{bmatrix} (A_{11} - A_{22}) & (A_{12} + A_{21}) & A_{33} \end{bmatrix}
 \end{aligned}$$



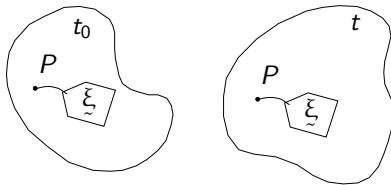
# KINEMATICS

[back to index](#)

# Kinematics

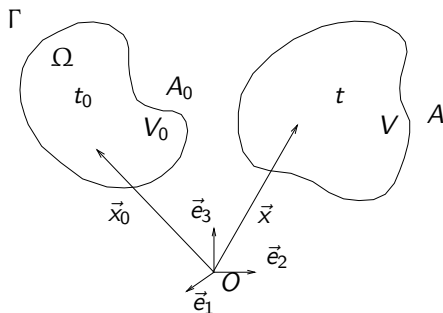


# Material coordinates



$$\tilde{\xi}^T = \begin{bmatrix} \xi_1 & \xi_2 & \xi_3 \end{bmatrix}$$

# Position vectors



undeformed configuration ( $t_0$ )

$$\vec{x}_0 = \vec{\chi}(\xi, t_0) = x_{01}\vec{e}_1 + x_{02}\vec{e}_2 + x_{03}\vec{e}_3$$

deformed configuration ( $t$ )

$$\vec{x} = \vec{\chi}(\xi, t) = x_1\vec{e}_1 + x_2\vec{e}_2 + x_3\vec{e}_3$$

# Euler-Lagrange

Euler : "observer" is fixed in space

$$a = \mathcal{A}_E(\vec{x}, t)$$

$$da = a_Q - a_P = \mathcal{A}_E(\vec{x} + d\vec{x}, t) - \mathcal{A}_E(\vec{x}, t) = d\vec{x} \cdot (\vec{\nabla} a) \Big|_t$$

$$\vec{\nabla} = \vec{e}_1 \frac{\partial}{\partial x_1} + \vec{e}_2 \frac{\partial}{\partial x_2} + \vec{e}_3 \frac{\partial}{\partial x_3}$$

Lagrange : "observer" follows the material

$$a = \mathcal{A}_L(\vec{x}_0, t)$$

$$da = a_Q - a_P = \mathcal{A}_L(\vec{x}_0 + d\vec{x}_0, t) - \mathcal{A}_L(\vec{x}_0, t) = d\vec{x}_0 \cdot (\vec{\nabla}_0 a) \Big|_t$$

$$\vec{\nabla}_0 = \vec{e}_1 \frac{\partial}{\partial x_{01}} + \vec{e}_2 \frac{\partial}{\partial x_{02}} + \vec{e}_3 \frac{\partial}{\partial x_{03}}$$

position vectors

$$\vec{\nabla} \vec{x} = \mathbf{I} \quad ; \quad \vec{\nabla}_0 \vec{x}_0 = \mathbf{I}$$

# Time derivatives

material time derivative

$$\frac{Da}{Dt} = \dot{a} = \lim_{\Delta t \rightarrow 0} \frac{1}{\Delta t} \{A(\vec{x}_0, t + \Delta t) - A(\vec{x}_0, t)\}$$

velocity of a material point

$$\vec{v} = \vec{v}(\vec{x}_0) = \dot{\vec{x}}$$

spatial time derivative

$$\frac{\delta a}{\delta t} = \lim_{\Delta t \rightarrow 0} \frac{1}{\Delta t} \{\mathcal{A}(\vec{x}, t + \Delta t) - \mathcal{A}(\vec{x}, t)\}$$

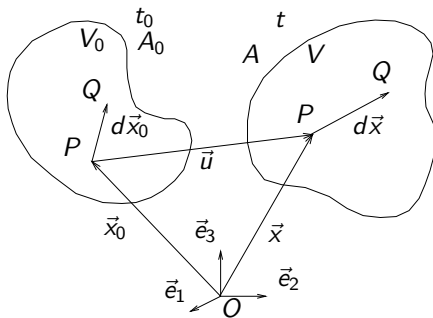
velocity *field*

$$\vec{v} = \vec{v}(\vec{x}, t)$$

## Relation $\vec{a}$ and $\frac{\delta \vec{a}}{\delta t}$

$$\begin{aligned}\frac{Da}{Dt} &= \lim_{\Delta t \rightarrow 0} \frac{1}{\Delta t} \{A(\vec{x}_0, t + \Delta t) - A(\vec{x}_0, t)\} \\&= \lim_{\Delta t \rightarrow 0} \frac{1}{\Delta t} \{\mathcal{A}(\vec{x} + d\vec{x}, t + \Delta t) - \mathcal{A}(\vec{x}, t)\} \\&= \lim_{\Delta t \rightarrow 0} \frac{1}{\Delta t} \{\mathcal{A}(\vec{x} + d\vec{x}, t + \Delta t) - \mathcal{A}(\vec{x}, t + \Delta t) + \\&\quad \mathcal{A}(\vec{x}, t + \Delta t) - \mathcal{A}(\vec{x}, t)\} \\&= \lim_{\Delta t \rightarrow 0} \frac{1}{\Delta t} \{d\vec{x} \cdot (\vec{\nabla} a) \Big|_{t+\Delta t} + \mathcal{A}(\vec{x}, t + \Delta t) - \mathcal{A}(\vec{x}, t)\} \\&= \lim_{\Delta t \rightarrow 0} \left\{ \frac{d\vec{x}}{\Delta t} \cdot (\vec{\nabla} a) \Big|_{t+\Delta t} \right\} + \lim_{\Delta t \rightarrow 0} \frac{1}{\Delta t} \{\mathcal{A}(\vec{x}, t + \Delta t) - \mathcal{A}(\vec{x}, t)\} \\&= \vec{v} \cdot (\vec{\nabla} a) + \frac{\delta a}{\delta t} \\&= (\text{convective time derivative}) + (\text{spatial time derivative}) \\&= (\text{material time derivative})\end{aligned}$$

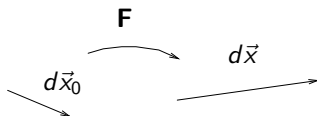
# Deformation



displacement :  $\vec{u} = \vec{x} - \vec{x}_0 = u_1\vec{e}_1 + u_2\vec{e}_2 + u_3\vec{e}_3$



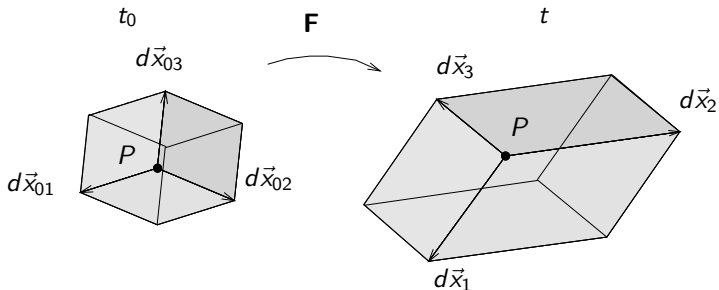
# Deformation tensor



$$\begin{aligned} d\vec{x} &= \mathbf{F} \cdot d\vec{x}_0 \\ &= \vec{X}(\vec{x}_0 + d\vec{x}_0, \mathbf{t}) - \vec{X}(\vec{x}_0, \mathbf{t}) = d\vec{x}_0 \cdot \left( \vec{\nabla}_0 \vec{x} \right) \\ &= \left( \vec{\nabla}_0 \vec{x} \right)^c \cdot d\vec{x}_0 = \mathbf{F} \cdot d\vec{x}_0 \end{aligned}$$

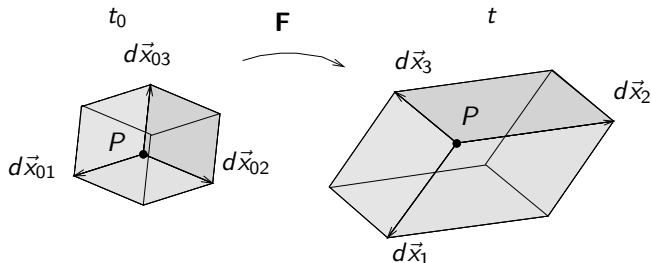
$$\mathbf{F} = \left( \vec{\nabla}_0 \vec{x} \right)^c = \left[ \left( \vec{\nabla}_0 \vec{x}_0 \right)^c + \left( \vec{\nabla}_0 \vec{u} \right)^c \right] = \mathbf{I} + \left( \vec{\nabla}_0 \vec{u} \right)^c$$

# Deformation tensor



$$d\vec{x}_1 = \mathbf{F} \cdot d\vec{x}_{01} \quad ; \quad d\vec{x}_2 = \mathbf{F} \cdot d\vec{x}_{02} \quad ; \quad d\vec{x}_3 = \mathbf{F} \cdot d\vec{x}_{03}$$

# Volume change



$$\begin{aligned} dV &= d\vec{x}_1 * d\vec{x}_2 \cdot d\vec{x}_3 \\ &= (\mathbf{F} \cdot d\vec{x}_{01}) * (\mathbf{F} \cdot d\vec{x}_{02}) \cdot (\mathbf{F} \cdot d\vec{x}_{03}) \\ &= \det(\mathbf{F}) (d\vec{x}_{01} * d\vec{x}_{02} \cdot d\vec{x}_{03}) \\ &= \det(\mathbf{F}) dV_0 \\ &= J dV_0 \end{aligned}$$

## Area change

$$\begin{aligned}dA \vec{n} &= d\vec{x}_1 * d\vec{x}_2 = (\mathbf{F} \cdot d\vec{x}_{01}) * (\mathbf{F} \cdot d\vec{x}_{02}) \\dA \vec{n} \cdot (\mathbf{F} \cdot d\vec{x}_{03}) &= (\mathbf{F} \cdot d\vec{x}_{01}) * (\mathbf{F} \cdot d\vec{x}_{02}) \cdot (\mathbf{F} \cdot d\vec{x}_{03}) \\&= \det(\mathbf{F})(d\vec{x}_{01} * d\vec{x}_{02}) \cdot d\vec{x}_{03} \quad \forall \quad d\vec{x}_{03} \rightarrow \\dA \vec{n} \cdot \mathbf{F} &= \det(\mathbf{F})(d\vec{x}_{01} * d\vec{x}_{02}) \\dA \vec{n} &= \det(\mathbf{F})(d\vec{x}_{01} * d\vec{x}_{02}) \cdot \mathbf{F}^{-1} \\&= \det(\mathbf{F}) dA_0 \vec{n}_0 \cdot \mathbf{F}^{-1} \\&= dA_0 \vec{n}_0 \cdot (\mathbf{F}^{-1} \det(\mathbf{F}))\end{aligned}$$

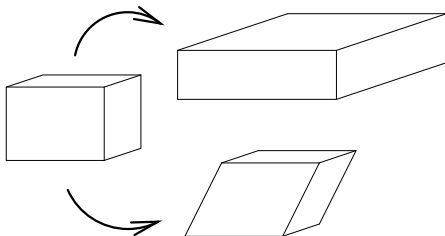
# Inverse deformation

$$J = \frac{dV}{dV_0} = \det(\mathbf{F}) > 0 \rightarrow \mathbf{F} \text{ regular} \rightarrow d\vec{x}_0 = \mathbf{F}^{-1} \cdot d\vec{x}$$

relation between gradient operators

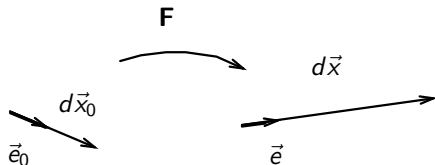
$$\mathbf{I} = \mathbf{F}^{-T} \cdot \mathbf{F}^T \rightarrow \left( \vec{\nabla} \vec{x} \right) = \mathbf{F}^{-T} \cdot \left( \vec{\nabla}_0 \vec{x} \right) \rightarrow \vec{\nabla} = \mathbf{F}^{-T} \cdot \vec{\nabla}_0$$

# Homogeneous deformation



$$\vec{\nabla}_0 \vec{x} = \mathbf{F}^c = \text{uniform tensor} \rightarrow$$
$$\vec{x} = (\vec{x}_0 \cdot \mathbf{F}^c) + \vec{t} = \mathbf{F} \cdot \vec{x}_0 + \vec{t}$$

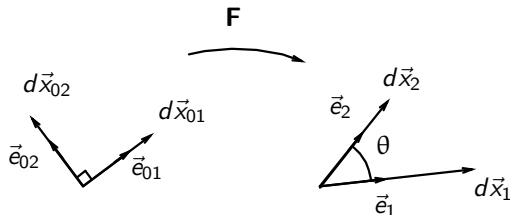
# Elongation and shear



elongation factor in initial  $\vec{e}_0$ -direction

$$\begin{aligned}\lambda^2(\vec{e}_0) &= \frac{d\vec{x}_1 \cdot d\vec{x}_1}{d\vec{x}_0 \cdot d\vec{x}_0} = \frac{d\vec{x}_0 \cdot \mathbf{F}^T \cdot \mathbf{F} \cdot d\vec{x}_0}{d\vec{x}_0 \cdot d\vec{x}_0} = \frac{\|d\vec{x}_0\|^2}{\|d\vec{x}_0\|^2} \left( \vec{e}_0 \cdot \mathbf{F}^T \cdot \mathbf{F} \cdot \vec{e}_0 \right) \\ &= \vec{e}_0 \cdot \mathbf{F}^T \cdot \mathbf{F} \cdot \vec{e}_0 = \vec{e}_0 \cdot \mathbf{C} \cdot \vec{e}_0\end{aligned}$$

# Elongation and shear

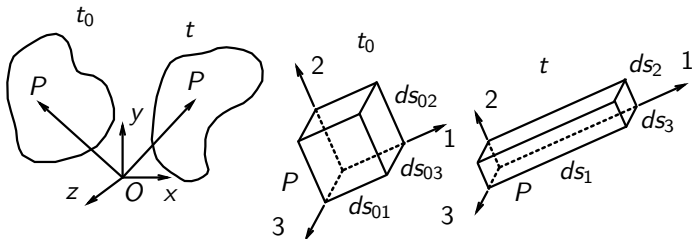


shear of initial  $(\vec{e}_{01}, \vec{e}_{02})$ -directions

$$\begin{aligned}
 \gamma(\vec{e}_{01}, \vec{e}_{02}) &= \sin\left(\frac{\pi}{2} - \theta\right) = \cos(\theta) = \frac{d\vec{x}_1 \cdot d\vec{x}_2}{\|d\vec{x}_1\| \|d\vec{x}_2\|} = \frac{d\vec{x}_{01} \cdot \mathbf{F}^T \cdot \mathbf{F} \cdot d\vec{x}_{02}}{\|d\vec{x}_1\| \|d\vec{x}_2\|} \\
 &= \frac{\|d\vec{x}_{01}\| \|d\vec{x}_{02}\| (\vec{e}_{01} \cdot \mathbf{F}^T \cdot \mathbf{F} \cdot \vec{e}_{02})}{\lambda(\vec{e}_{01}) \|d\vec{x}_{01}\| \lambda(\vec{e}_{02}) \|d\vec{x}_{02}\|} = \frac{\vec{e}_{01} \cdot \mathbf{F}^T \cdot \mathbf{F} \cdot \vec{e}_{02}}{\lambda(\vec{e}_{01}) \lambda(\vec{e}_{02})} \\
 &= \frac{\vec{e}_{01} \cdot \mathbf{C} \cdot \vec{e}_{02}}{\lambda(\vec{e}_{01}) \lambda(\vec{e}_{02})}
 \end{aligned}$$



# Principal directions of deformation



$$\lambda_1 = \frac{ds_1}{ds_{01}} \quad ; \quad \lambda_2 = \frac{ds_2}{ds_{02}} \quad ; \quad \lambda_3 = \frac{ds_3}{ds_{03}} \quad ; \quad \gamma_{12} = \gamma_{23} = \gamma_{31} = 0$$

$$J = \frac{dV}{dV_0} = \frac{ds_1 ds_2 ds_3}{ds_{01} ds_{02} ds_{03}} = \lambda_1 \lambda_2 \lambda_3$$

# Strains

$$\varepsilon = f(\lambda)$$

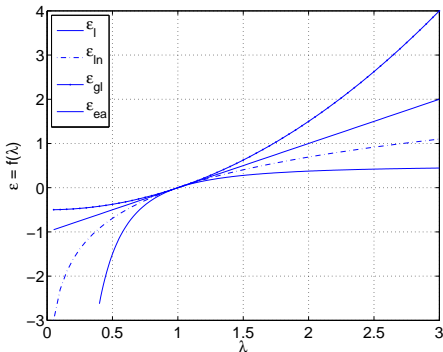
- $f(\lambda = 1) = 0$
- $\lim_{\lambda \rightarrow 1} f(\lambda) = \lambda - 1$
- $f(\lambda)$  monotonic increasing
- $f(\lambda)$  differentiable

linear  $\varepsilon_l = \lambda - 1$

logarithmic  $\varepsilon_{ln} = \ln(\lambda)$

Green-Lagrange  $\varepsilon_{gl} = \frac{1}{2}(\lambda^2 - 1)$

Euler-Almansi  $\varepsilon_{ea} = \frac{1}{2} \left( 1 - \frac{1}{\lambda^2} \right)$



# Strain tensor

$$\frac{1}{2} \{ \lambda^2(\vec{e}_{01}) - 1 \} = \vec{e}_{01} \cdot \left\{ \frac{1}{2} (\mathbf{F}^T \cdot \mathbf{F} - \mathbf{I}) \right\} \cdot \vec{e}_{01} = \vec{e}_{01} \cdot \mathbf{E} \cdot \vec{e}_{01}$$

$$\gamma(\vec{e}_{01}, \vec{e}_{02}) = \frac{\vec{e}_{01} \cdot (\mathbf{F}^T \cdot \mathbf{F} - \mathbf{I}) \cdot \vec{e}_{02}}{\lambda(\vec{e}_{01})\lambda(\vec{e}_{02})} = \left[ \frac{2}{\lambda(\vec{e}_{01})\lambda(\vec{e}_{02})} \right] \vec{e}_{01} \cdot \mathbf{E} \cdot \vec{e}_{02}$$

$$\left. \begin{aligned} \mathbf{E} &= \frac{1}{2} (\mathbf{F}^T \cdot \mathbf{F} - \mathbf{I}) \\ \mathbf{F} &= (\vec{\nabla}_0 \vec{x})^T = \mathbf{I} + (\vec{\nabla}_0 \vec{u})^T \end{aligned} \right\} \rightarrow \left. \begin{aligned} \mathbf{E} &= \frac{1}{2} \left[ \left\{ \mathbf{I} + (\vec{\nabla}_0 \vec{u}) \right\} \cdot \left\{ \mathbf{I} + (\vec{\nabla}_0 \vec{u})^T \right\} - \mathbf{I} \right] \\ &= \frac{1}{2} \left[ (\vec{\nabla}_0 \vec{u})^T + (\vec{\nabla}_0 \vec{u}) + (\vec{\nabla}_0 \vec{u}) \cdot (\vec{\nabla}_0 \vec{u}) \right] \end{aligned} \right\}$$

# Right Cauchy-Green deformation tensor

$$\mathbf{C} = \mathbf{F}^c \cdot \mathbf{F}$$

1. symmetric  $\mathbf{C}^c = \mathbf{C}$
2. positive definite

$$\vec{a} \cdot \mathbf{C} \cdot \vec{a} = \vec{a} \cdot \mathbf{F}^c \cdot \mathbf{F} \cdot \vec{a} = (\mathbf{F} \cdot \vec{a}) \cdot (\mathbf{F} \cdot \vec{a})$$

$$\mathbf{F} \text{ is regular} \quad \rightarrow \quad \mathbf{F} \cdot \vec{a} \neq \vec{0} \quad \text{if} \quad \vec{a} \neq \vec{0} \quad \rightarrow$$

$$\vec{a} \cdot \mathbf{C} \cdot \vec{a} > 0 \quad \forall \quad \vec{a} \neq \vec{0}$$

3.  $\left. \begin{array}{l} \text{eigenvalues and eigenvectors real} \\ \text{eigenvalues positive} \\ \text{eigenvectors} \perp \text{ (choice)} \end{array} \right\} \rightarrow \text{spectral representation}$

$$\mathbf{C} = \mu_1 \vec{m}_1 \vec{m}_1 + \mu_2 \vec{m}_2 \vec{m}_2 + \mu_3 \vec{m}_3 \vec{m}_3$$

# Eigenvectors and eigenvalues

$$\mathbf{C} = \mu_1 \vec{m}_1 \vec{m}_1 + \mu_2 \vec{m}_2 \vec{m}_2 + \mu_3 \vec{m}_3 \vec{m}_3$$

$$\lambda(\vec{e}_0) = \sqrt{\vec{e}_0 \cdot \mathbf{C} \cdot \vec{e}_0} \quad ; \quad \gamma(\vec{e}_{01}, \vec{e}_{02}) = \frac{\vec{e}_{01} \cdot \mathbf{C} \cdot \vec{e}_{02}}{\sqrt{\vec{e}_{01} \cdot \mathbf{C} \cdot \vec{e}_{01}} \sqrt{\vec{e}_{02} \cdot \mathbf{C} \cdot \vec{e}_{02}}} \rightarrow$$

$$\mathbf{C} = \mu_1 \vec{n}_{01} \vec{n}_{01} + \mu_2 \vec{n}_{02} \vec{n}_{02} + \mu_3 \vec{n}_{03} \vec{n}_{03}$$

$$\lambda(\vec{n}_{01}) = \sqrt{\vec{n}_{01} \cdot \mathbf{C} \cdot \vec{n}_{01}} = \sqrt{\mu_1} \quad ; \quad \gamma(\vec{n}_{01}, \vec{n}_{02}) = \frac{\vec{n}_{01} \cdot \mathbf{C} \cdot \vec{n}_{02}}{\sqrt{\vec{n}_{01} \cdot \mathbf{C} \cdot \vec{n}_{01}} \sqrt{\vec{n}_{02} \cdot \mathbf{C} \cdot \vec{n}_{02}}} = 0$$

$$\mathbf{C} = \lambda_1^2 \vec{n}_{01} \vec{n}_{01} + \lambda_2^2 \vec{n}_{02} \vec{n}_{02} + \lambda_3^2 \vec{n}_{03} \vec{n}_{03}$$

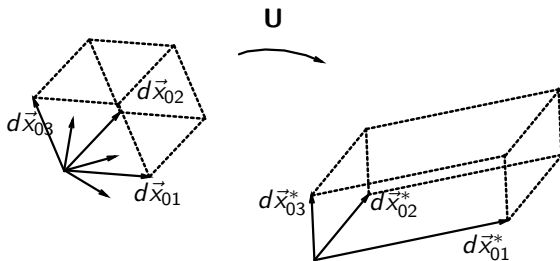
# Right stretch tensor

$$\mathbf{U} = \sqrt{\mathbf{C}} = \lambda_1 \vec{n}_{01} \vec{n}_{01} + \lambda_2 \vec{n}_{02} \vec{n}_{02} + \lambda_3 \vec{n}_{03} \vec{n}_{03}$$

## properties

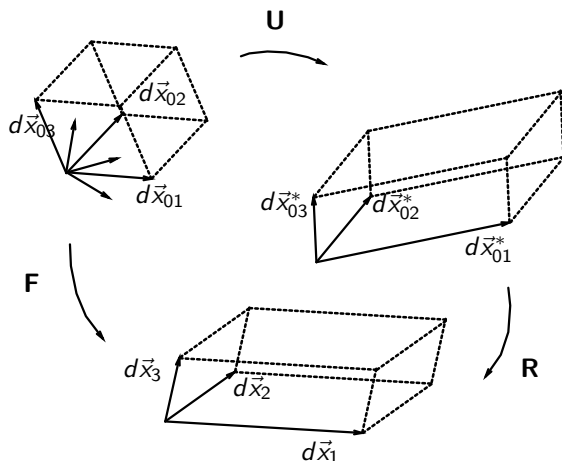
1. symmetric :  $\mathbf{U}^c = \mathbf{U}$
2. pos. def. :  $\vec{a} \cdot \mathbf{U} \cdot \vec{a} > 0 \quad \forall \quad \vec{a}$
3. regular :  $\mathbf{U}^{-1} = \frac{1}{\lambda_1} \vec{n}_{01} \vec{n}_{01} + \frac{1}{\lambda_2} \vec{n}_{02} \vec{n}_{02} + \frac{1}{\lambda_3} \vec{n}_{03} \vec{n}_{03}$
4.  $\det(\mathbf{C}) = \det(\mathbf{U} \cdot \mathbf{U}) = \det(\mathbf{F}^c \cdot \mathbf{F}) = \det^2(\mathbf{F}) \rightarrow$   
 $\det(\mathbf{U}) = \lambda_1 \lambda_2 \lambda_3 = \det(\mathbf{F}) = J$

# Stretch tensor transformation



$$d\vec{x}_{01}^* = \mathbf{U} \cdot d\vec{x}_{01} \quad ; \quad d\vec{x}_{02}^* = \mathbf{U} \cdot d\vec{x}_{02} \quad ; \quad d\vec{x}_{03}^* = \mathbf{U} \cdot d\vec{x}_{03}$$

# Total transformation



$$\left. \begin{aligned} d\vec{x}_{01}^* &= \mathbf{U} \cdot d\vec{x}_{01} \rightarrow d\vec{x}_{01} = \mathbf{U}^{-1} \cdot d\vec{x}_{01}^* \\ d\vec{x}_1 &= \mathbf{F} \cdot d\vec{x}_{01} \end{aligned} \right\} \rightarrow$$

$$d\vec{x}_1 = \mathbf{F} \cdot \mathbf{U}^{-1} \cdot d\vec{x}_{01}^* = \mathbf{R} \cdot d\vec{x}_{01}^* \rightarrow \mathbf{R} = \mathbf{F} \cdot \mathbf{U}^{-1}$$



# Rotation tensor

$$\mathbf{R} = \mathbf{F} \cdot \mathbf{U}^{-1}$$

1.

$$\begin{aligned}\mathbf{R}^c \cdot \mathbf{R} &= \mathbf{U}^{-c} \cdot \mathbf{F}^c \cdot \mathbf{F} \cdot \mathbf{U}^{-1} \\ &= \mathbf{U}^{-c} \cdot \mathbf{U} \cdot \mathbf{U} \cdot \mathbf{U}^{-1} \\ &= \mathbf{U}^{-c} \cdot \mathbf{U}^c \cdot \mathbf{U} \cdot \mathbf{U}^{-1} \\ &= \mathbf{I} \quad \rightarrow \quad \mathbf{R} \text{ is orthogonal}\end{aligned}$$

2.

$$\begin{aligned}\det(\mathbf{R}) &= \det(\mathbf{F} \cdot \mathbf{U}^{-1}) \\ &= \det(\mathbf{U}) \det(\mathbf{U}^{-1}) = \det(\mathbf{U} \cdot \mathbf{U}^{-1}) \\ &= \det(\mathbf{I}) = 1 \quad \rightarrow \quad \mathbf{R} \text{ is rotation tensor}\end{aligned}$$

# Right polar decomposition

$$\mathbf{F} = \mathbf{R} \cdot \mathbf{U}$$

- $\mathbf{F}$  known
- calculate  $\mathbf{C} = \mathbf{F}^c \cdot \mathbf{F}$
- calculate  $\lambda_i$  en  $\vec{n}_{0i}$
- $\mathbf{U}$  known
- calculate  $\mathbf{U}^{-1}$
- calculate  $\mathbf{R} = \mathbf{F} \cdot \mathbf{U}^{-1}$

# Strain tensors

stretch ratio

$$\lambda(\vec{e}_0) = \sqrt{\vec{e}_0 \cdot \mathbf{C} \cdot \vec{e}_0}$$

strain tensor

$$\boldsymbol{\varepsilon}$$

strain measure

$$\varepsilon(\vec{e}_0) = \vec{e}_0 \cdot \boldsymbol{\varepsilon} \cdot \vec{e}_0 = f(\lambda(\vec{e}_0))$$

shear measure

$$\gamma(\vec{e}_{01}, \vec{e}_{02}) = \vec{e}_{01} \cdot \boldsymbol{\varepsilon} \cdot \vec{e}_{02}$$

# Linear strain tensor

$$\boldsymbol{\mathcal{E}} = \mathbf{U} - \mathbf{I}$$

$$\begin{aligned}\vec{e}_0 \cdot \boldsymbol{\mathcal{E}} \cdot \vec{e}_0 &= \vec{e}_0 \cdot \mathbf{U} \cdot \vec{e}_0 - \vec{e}_0 \cdot \mathbf{I} \cdot \vec{e}_0 \\ &= \vec{e}_0 \cdot \mathbf{U} \cdot \vec{e}_0 - 1 \\ &\neq \lambda(\vec{e}_0) - 1\end{aligned}$$

$$\begin{aligned}\vec{n}_{0i} \cdot \boldsymbol{\mathcal{E}} \cdot \vec{n}_{0i} &= \vec{n}_{0i} \cdot \mathbf{U} \cdot \vec{n}_{0i} - 1 \\ &= \lambda(\vec{n}_{0i}) - 1 \\ &= \lambda_i - 1\end{aligned}$$

# Logarithmic strain tensor

$$\mathbf{\Lambda} = \ln(\mathbf{U})$$

$$\begin{aligned}\vec{e}_0 \cdot \mathbf{\Lambda} \cdot \vec{e}_0 &= \vec{e}_0 \cdot \ln(\mathbf{U}) \cdot \vec{e}_0 \\ &\neq \ln(\lambda(\vec{e}_0))\end{aligned}$$

$$\begin{aligned}\vec{n}_{0i} \cdot \mathbf{\Lambda} \cdot \vec{n}_{0i} &= \vec{n}_{0i} \cdot \ln(\mathbf{U}) \cdot \vec{n}_{0i} \\ &= \ln(\lambda(\vec{n}_{0i})) \\ &= \ln(\lambda_i)\end{aligned}$$

# Green-Lagrange strain tensor

$$\mathbf{E} = \frac{1}{2} (\mathbf{C} - \mathbf{I})$$

$$\begin{aligned}\vec{e}_0 \cdot \mathbf{E} \cdot \vec{e}_0 &= \frac{1}{2} (\vec{e}_0 \cdot \mathbf{C} \cdot \vec{e}_0 - 1) \\ &= \frac{1}{2} (\lambda^2(\vec{e}_0) - 1)\end{aligned}$$

# Infinitesimal linear strain tensor

$$\begin{aligned}\mathbf{E} &= \frac{1}{2} (\mathbf{F}^c \cdot \mathbf{F} - \mathbf{I}) \\ &= \frac{1}{2} \left\{ (\vec{\nabla}_0 \vec{u}) + (\vec{\nabla}_0 \vec{u})^c + (\vec{\nabla}_0 \vec{u}) \cdot (\vec{\nabla}_0 \vec{u})^c \right\}\end{aligned}$$

linearisation  $\rightarrow$  infinitesimal strain tensor

$$\begin{aligned}\boldsymbol{\varepsilon} &= \frac{1}{2} \left\{ (\vec{\nabla}_0 \vec{u}) + (\vec{\nabla}_0 \vec{u})^c \right\} \\ &= \frac{1}{2} (\mathbf{F} + \mathbf{F}^c) - \mathbf{I} \\ &= \frac{1}{2} \left\{ (\vec{\nabla} \vec{u}) + (\vec{\nabla} \vec{u})^c \right\}\end{aligned}$$

only correct for small strains AND small rotations

# Deformation rate

$$\begin{aligned} d\dot{\vec{x}} &= \dot{\vec{F}} \cdot d\vec{x}_0 = \dot{\vec{F}} \cdot \vec{F}^{-1} \cdot d\vec{x} = \mathbf{L} \cdot d\vec{x} = (\vec{\nabla} \vec{v})^c \cdot d\vec{x} \\ &= \frac{1}{2}\{\mathbf{L} + \mathbf{L}^c\} \cdot d\vec{x} + \frac{1}{2}\{\mathbf{L} - \mathbf{L}^c\} \cdot d\vec{x} \\ &= \mathbf{D} \cdot d\vec{x} + \mathbf{\Omega} \cdot d\vec{x} \end{aligned}$$

velocity gradient tensor	$\mathbf{L}$
deformation rate tensor	$\mathbf{D}$
rotation rate tensor or spin tensor	$\mathbf{\Omega}$



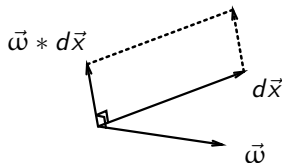
# Spin tensor

$$\boldsymbol{\Omega} = \frac{1}{2} \left\{ \dot{\mathbf{F}} \cdot \mathbf{F}^{-1} - (\dot{\mathbf{F}} \cdot \mathbf{F}^{-1})^c \right\} = \frac{1}{2} \left\{ \left( \vec{\nabla} \vec{v} \right)^c - \left( \vec{\nabla} \vec{v} \right) \right\}$$

$\boldsymbol{\Omega}$  = skewsymmetric  $\rightarrow$

$\boldsymbol{\Omega} \cdot d\vec{x} = \vec{\omega} * d\vec{x} = \text{velocity} \perp d\vec{x} = \text{rotation rate}$

$\vec{\omega}$  : axial vector



# Axial vector

$$\vec{q} \cdot \boldsymbol{\Omega} \cdot \vec{q} = \vec{q} \cdot \boldsymbol{\Omega}^c \cdot \vec{q} = - \vec{q} \cdot \boldsymbol{\Omega} \cdot \vec{q} \quad \rightarrow$$

$$\vec{q} \cdot \boldsymbol{\Omega} \cdot \vec{q} = 0 \quad \rightarrow$$

$$\boldsymbol{\Omega} \cdot \vec{q} = \vec{p} \quad \rightarrow$$

$$\vec{q} \cdot \vec{p} = 0 \quad \rightarrow$$

$$\vec{q} \perp \vec{p} \quad \rightarrow$$

$$\exists \quad \vec{\omega} \quad \text{zdd} \quad \vec{p} = \vec{\omega} * \vec{q} \quad \rightarrow$$

$$\boldsymbol{\Omega} \cdot \vec{q} = \vec{\omega} * \vec{q}$$

## Axial vector components

$$\boldsymbol{\Omega} \cdot \vec{q} = \vec{\omega} * \vec{q} \quad \forall \quad \vec{q}$$

$$\boldsymbol{\Omega} \cdot \vec{q} = \tilde{\vec{e}}^T \begin{bmatrix} \Omega_{11} & \Omega_{12} & \Omega_{13} \\ \Omega_{21} & \Omega_{22} & \Omega_{23} \\ \Omega_{31} & \Omega_{32} & \Omega_{33} \end{bmatrix} \begin{bmatrix} q_1 \\ q_2 \\ q_3 \end{bmatrix} = \tilde{\vec{e}}^T \begin{bmatrix} \Omega_{11}q_1 + \Omega_{12}q_2 + \Omega_{13}q_3 \\ \Omega_{21}q_1 + \Omega_{22}q_2 + \Omega_{23}q_3 \\ \Omega_{31}q_1 + \Omega_{32}q_2 + \Omega_{33}q_3 \end{bmatrix}$$

$$\begin{aligned} \vec{\omega} * \vec{q} &= (\omega_1 \vec{e}_1 + \omega_2 \vec{e}_2 + \omega_3 \vec{e}_3) * (q_1 \vec{e}_1 + q_2 \vec{e}_2 + q_3 \vec{e}_3) \\ &= \omega_1 q_2 (\vec{e}_3) + \omega_1 q_3 (-\vec{e}_2) + \omega_2 q_1 (-\vec{e}_3) + \omega_2 q_3 (\vec{e}_1) + \\ &\quad \omega_3 q_1 (\vec{e}_2) + \omega_3 q_2 (-\vec{e}_1) \\ &= [\vec{e}_1 \ \vec{e}_2 \ \vec{e}_3] \begin{bmatrix} \omega_2 q_3 - \omega_3 q_2 \\ \omega_3 q_1 - \omega_1 q_3 \\ \omega_1 q_2 - \omega_2 q_1 \end{bmatrix} \end{aligned}$$

$$\boldsymbol{\Omega} \cdot \vec{q} = \vec{\omega} * \vec{q} \quad \forall \vec{q} \quad \rightarrow \quad \underline{\underline{\Omega}} = \begin{bmatrix} 0 & -\omega_3 & \omega_2 \\ \omega_3 & 0 & -\omega_1 \\ -\omega_2 & \omega_1 & 0 \end{bmatrix}$$

# Deformation rate tensor

$$\mathbf{D} = \frac{1}{2} \left\{ \dot{\mathbf{F}} \cdot \mathbf{F}^{-1} + (\dot{\mathbf{F}} \cdot \mathbf{F}^{-1})^c \right\} = \left\{ \left( \vec{\nabla} \vec{v} \right)^c + \left( \vec{\nabla} \vec{v} \right) \right\}$$

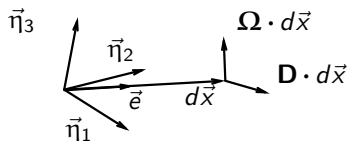
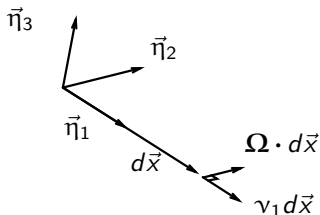
$$\mathbf{D} = \mathbf{D}^c \rightarrow \mathbf{D} = \nu_1 \vec{\eta}_1 \vec{\eta}_1 + \nu_2 \vec{\eta}_2 \vec{\eta}_2 + \nu_3 \vec{\eta}_3 \vec{\eta}_3$$

1.: vector  $d\vec{x}$  along  $\vec{\eta}_1$  :  $d\vec{x} = dx_1 \vec{\eta}_1$

$$\mathbf{D} \cdot d\vec{x} = dx_1 \mathbf{D} \cdot \vec{\eta}_1 = dx_1 \nu_1 \vec{\eta}_1 = \nu_1 d\vec{x}$$

2.: random vector :  $d\vec{x} = dx_1 \vec{\eta}_1 + dx_2 \vec{\eta}_2 + dx_3 \vec{\eta}_3$

$$\mathbf{D} \cdot d\vec{x} = dx_1 \nu_1 \vec{\eta}_1 + dx_2 \nu_2 \vec{\eta}_2 + dx_3 \nu_3 \vec{\eta}_3$$



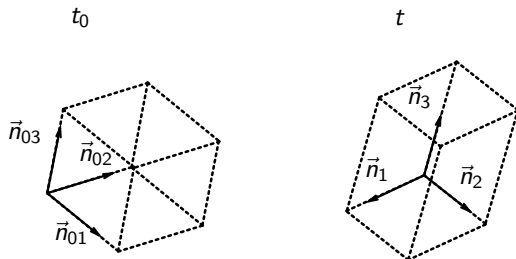
# Elongation rate

$$\lambda^2 = \vec{e}_0 \cdot \mathbf{C} \cdot \vec{e}_0 \quad \rightarrow \quad \frac{D}{Dt}(\lambda^2) = \frac{D}{Dt}(\vec{e}_0 \cdot \mathbf{C} \cdot \vec{e}_0) \quad \rightarrow$$

$$\begin{aligned} 2\lambda\dot{\lambda} &= \vec{e}_0 \cdot \frac{D}{Dt}(\mathbf{C}) \cdot \vec{e}_0 = \vec{e}_0 \cdot \frac{D}{Dt}(\mathbf{F}^c \cdot \mathbf{F}) \cdot \vec{e}_0 \\ &= \vec{e}_0 \cdot \{\dot{\mathbf{F}}^c \cdot \mathbf{F} + \mathbf{F}^c \cdot \dot{\mathbf{F}}\} \cdot \vec{e}_0 \\ &= \vec{e}_0 \cdot \mathbf{F}^c \cdot \{\mathbf{F}^{-c} \cdot \dot{\mathbf{F}}^c + \dot{\mathbf{F}} \cdot \mathbf{F}^{-1}\} \cdot \mathbf{F} \cdot \vec{e}_0 \\ &= (\mathbf{F} \cdot \vec{e}_0) \cdot \{(\dot{\mathbf{F}} \cdot \mathbf{F}^{-1})^c + \dot{\mathbf{F}} \cdot \mathbf{F}^{-1}\} \cdot (\mathbf{F} \cdot \vec{e}_0) \\ &= (\lambda \vec{e}) \cdot (2 \mathbf{D}) \cdot (\lambda \vec{e}) \quad \rightarrow \end{aligned}$$

$$\frac{\dot{\lambda}}{\lambda} = \vec{e} \cdot \mathbf{D} \cdot \vec{e}$$

# Volume change rate



$$\begin{aligned}\text{tr}(\mathbf{D}) &= \vec{n}_1 \cdot \mathbf{D} \cdot \vec{n}_1 + \vec{n}_2 \cdot \mathbf{D} \cdot \vec{n}_2 + \vec{n}_3 \cdot \mathbf{D} \cdot \vec{n}_3 = \frac{\dot{\lambda}_1}{\lambda_1} + \frac{\dot{\lambda}_2}{\lambda_2} + \frac{\dot{\lambda}_3}{\lambda_3} \\ &= \frac{D}{Dt} \{ \ln(\lambda_1) + \ln(\lambda_2) + \ln(\lambda_3) \} = \frac{D}{Dt} \{ \ln(\lambda_1 \lambda_2 \lambda_3) \} \\ &= \frac{D}{Dt} [ \ln \{ \det(\mathbf{U}) \} ] = \frac{D}{Dt} [ \ln \{ \det(\mathbf{F}) \} ] = \frac{D}{Dt} \{ \ln(J) \} = \frac{\dot{J}}{J} \rightarrow\end{aligned}$$

$$\dot{J} = J \text{tr}(\mathbf{D}) = J \left( \vec{\nabla} \cdot \vec{v} \right)$$

## Area change rate

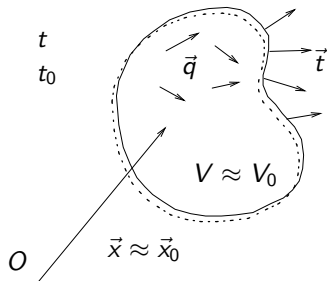
$$\begin{aligned}\frac{D}{Dt}(dA \vec{n}) &= \frac{D}{Dt} \{ \det(\mathbf{F}) dA_0 \vec{n}_0 \cdot \mathbf{F}^{-1} \} \\&= \frac{D}{Dt} \{ \det(\mathbf{F}) \} dA_0 \vec{n}_0 \cdot \mathbf{F}^{-1} + \det(\mathbf{F}) dA_0 \vec{n}_0 \cdot \dot{\mathbf{F}}^{-1} \\&= \dot{J} dA_0 \vec{n}_0 \cdot \mathbf{F}^{-1} - J dA_0 \vec{n}_0 \cdot \mathbf{F}^{-1} \cdot \mathbf{L} \\&= \text{tr}(\mathbf{L}) J dA_0 \vec{n}_0 \cdot \mathbf{F}^{-1} - J dA_0 \vec{n}_0 \cdot \mathbf{F}^{-1} \cdot \mathbf{L} \\&= J \text{tr}(\mathbf{L}) \mathbf{F}^{-c} \cdot dA_0 \vec{n}_0 - J \mathbf{L}^c \cdot \mathbf{F}^{-c} \cdot dA_0 \vec{n}_0 \\&= J (\text{tr}(\mathbf{L}) \mathbf{I} - \mathbf{L}^c) \cdot \mathbf{F}^{-c} \cdot dA_0 \vec{n}_0 \\&= (\text{tr}(\mathbf{L}) \mathbf{I} - \mathbf{L}^c) dA \vec{n}\end{aligned}$$

# SMALL (LINEAR) DEFORMATION

[back to index](#)



# Linear deformation

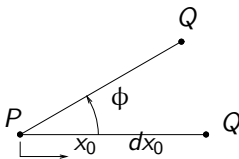


$$\mathbf{E} = \frac{1}{2} \left[ \left( \vec{\nabla}_0 \vec{u} \right)^T + \left( \vec{\nabla}_0 \vec{u} \right) + \left( \vec{\nabla}_0 \vec{u} \right) \cdot \left( \vec{\nabla}_0 \vec{u} \right)^T \right] \quad \left. \vphantom{\mathbf{E}} \right\} \rightarrow$$

small deformation  $\rightarrow \left( \vec{\nabla}_0 \vec{u} \right)^T = \mathbf{F} - \mathbf{I} \approx \mathbf{0}$

$$\mathbf{E} \approx \frac{1}{2} \left[ \left( \vec{\nabla}_0 \vec{u} \right)^T + \left( \vec{\nabla}_0 \vec{u} \right) \right] \approx \frac{1}{2} \left[ \left( \vec{\nabla} \vec{u} \right)^T + \left( \vec{\nabla} \vec{u} \right) \right] = \boldsymbol{\varepsilon} \quad \text{symm!}$$

# Rigid rotation



$$\left. \begin{aligned} u &= u_Q = -[dx_0 - dx_0 \cos(\phi)] = [\cos(\phi) - 1]dx_0 \\ v &= v_Q = [\sin(\phi)]dx_0 \end{aligned} \right\} \rightarrow$$

$$\frac{\partial u}{\partial x_0} = \cos(\phi) - 1 \quad ; \quad \frac{\partial v}{\partial x_0} = \sin(\phi) \quad \rightarrow$$

$$\varepsilon_{gl} = \frac{\partial u}{\partial x_0} + \frac{1}{2} \left( \frac{\partial u}{\partial x_0} \right)^2 + \frac{1}{2} \left( \frac{\partial v}{\partial x_0} \right)^2 = 0$$

$$\varepsilon_I = \frac{\partial u}{\partial x_0} = \cos(\phi) - 1 \neq 0 \quad !!$$

# Elongational, shear and volume strain

elong. strain

$$\begin{aligned}\frac{1}{2} (\lambda^2(\vec{e}_{01}) - 1) &= \vec{e}_{01} \cdot \mathbf{E} \cdot \vec{e}_{01} \\ &\downarrow \\ \lambda(\vec{e}_{01}) - 1 &= \vec{e}_{01} \cdot \boldsymbol{\varepsilon} \cdot \vec{e}_{01}\end{aligned}$$

shear strain

$$\begin{aligned}\gamma(\vec{e}_{01}, \vec{e}_{02}) = \sin\left(\frac{\pi}{2} - \theta\right) &= \left(\frac{2}{\lambda(\vec{e}_{01})\lambda(\vec{e}_{02})}\right) \vec{e}_{01} \cdot \mathbf{E} \cdot \vec{e}_{02} \\ &\downarrow \\ \frac{\pi}{2} - \theta &= 2 \vec{e}_{01} \cdot \boldsymbol{\varepsilon} \cdot \vec{e}_{02}\end{aligned}$$

volume change

$$\begin{aligned}J = \frac{dV}{dV_0} &= \lambda_1 \lambda_2 \lambda_3 = (\varepsilon_1 + 1)(\varepsilon_2 + 1)(\varepsilon_3 + 1) \\ &\downarrow \\ J &= \varepsilon_1 + \varepsilon_2 + \varepsilon_3 + 1 = \text{tr}(\boldsymbol{\varepsilon}) + 1\end{aligned}$$

volume strain

$$J - 1 = \text{tr}(\boldsymbol{\varepsilon})$$

# Linear strain matrix

$$\underline{\varepsilon} = \begin{bmatrix} \varepsilon_{11} & \varepsilon_{12} & \varepsilon_{13} \\ \varepsilon_{21} & \varepsilon_{22} & \varepsilon_{23} \\ \varepsilon_{31} & \varepsilon_{32} & \varepsilon_{33} \end{bmatrix} \quad \text{with} \quad \begin{cases} \varepsilon_{21} = \varepsilon_{12} \\ \varepsilon_{32} = \varepsilon_{23} \\ \varepsilon_{31} = \varepsilon_{13} \end{cases}$$

principal strain matrix

$$\underline{\varepsilon} = \begin{bmatrix} \varepsilon_1 & 0 & 0 \\ 0 & \varepsilon_2 & 0 \\ 0 & 0 & \varepsilon_3 \end{bmatrix}$$

spectral form

$$\underline{\varepsilon} = \varepsilon_1 \vec{n}_1 \vec{n}_1 + \varepsilon_2 \vec{n}_2 \vec{n}_2 + \varepsilon_3 \vec{n}_3 \vec{n}_3$$

# Linear strain : Cartesian components

gradient operator	$\vec{\nabla} = \vec{e}_x \frac{\partial}{\partial x} + \vec{e}_y \frac{\partial}{\partial y} + \vec{e}_z \frac{\partial}{\partial z}$
displacement vector	$\vec{u} = u_x \vec{e}_x + u_y \vec{e}_y + u_z \vec{e}_z$
linear strain tensor	$\underline{\varepsilon} = \frac{1}{2} \left\{ (\vec{\nabla} \vec{u})^c + (\vec{\nabla} \vec{u}) \right\} = \vec{e}^T \underline{\varepsilon} \vec{e}$

$$\underline{\varepsilon} = \begin{bmatrix} \varepsilon_{xx} & \varepsilon_{xy} & \varepsilon_{xz} \\ \varepsilon_{yx} & \varepsilon_{yy} & \varepsilon_{yz} \\ \varepsilon_{zx} & \varepsilon_{zy} & \varepsilon_{zz} \end{bmatrix}$$
$$= \frac{1}{2} \begin{bmatrix} 2u_{x,x} & u_{x,y} + u_{y,x} & u_{x,z} + u_{z,x} \\ u_{y,x} + u_{x,y} & 2u_{y,y} & u_{y,z} + u_{z,y} \\ u_{z,x} + u_{x,z} & u_{z,y} + u_{y,z} & 2u_{z,z} \end{bmatrix}$$

## Linear strain : cylindrical components

gradient operator	$\vec{\nabla} = \vec{e}_r \frac{\partial}{\partial r} + \vec{e}_t \frac{1}{r} \frac{\partial}{\partial \theta} + \vec{e}_z \frac{\partial}{\partial z}$
displacement vector	$\vec{u} = u_r \vec{e}_r(\theta) + u_t \vec{e}_t(\theta) + u_z \vec{e}_z$
linear strain tensor	$\underline{\varepsilon} = \frac{1}{2} \left\{ (\vec{\nabla} \vec{u})^c + (\vec{\nabla} \vec{u}) \right\} = \tilde{\vec{e}}^T \underline{\varepsilon} \tilde{\vec{e}}$

$$\underline{\varepsilon} = \begin{bmatrix} \varepsilon_{rr} & \varepsilon_{rt} & \varepsilon_{rz} \\ \varepsilon_{tr} & \varepsilon_{tt} & \varepsilon_{tz} \\ \varepsilon_{zr} & \varepsilon_{zt} & \varepsilon_{zz} \end{bmatrix}$$
$$= \frac{1}{2} \begin{bmatrix} 2u_{r,r} & \frac{1}{r}(u_{r,t} - u_t) + u_{t,r} & u_{r,z} + u_{z,r} \\ \frac{1}{r}(u_{r,t} - u_t) + u_{t,r} & 2\frac{1}{r}(u_r + u_{t,t}) & \frac{1}{r}u_{z,t} + u_{t,z} \\ u_{z,r} + u_{r,z} & \frac{1}{r}u_{z,t} + u_{t,z} & 2u_{z,z} \end{bmatrix}$$

## Compatibility relations

$$\frac{\partial^2 \varepsilon_{xx}}{\partial y^2} + \frac{\partial^2 \varepsilon_{yy}}{\partial x^2} = 2 \frac{\partial^2 \varepsilon_{xy}}{\partial x \partial y}$$

$$\frac{\partial^2 \varepsilon_{yy}}{\partial z^2} + \frac{\partial^2 \varepsilon_{zz}}{\partial y^2} = 2 \frac{\partial^2 \varepsilon_{yz}}{\partial y \partial z}$$

$$\frac{\partial^2 \varepsilon_{zz}}{\partial x^2} + \frac{\partial^2 \varepsilon_{xx}}{\partial z^2} = 2 \frac{\partial^2 \varepsilon_{zx}}{\partial z \partial x}$$

$$\frac{\partial^2 \varepsilon_{xx}}{\partial y \partial z} + \frac{\partial^2 \varepsilon_{yz}}{\partial x^2} = \frac{\partial^2 \varepsilon_{xz}}{\partial x \partial y} + \frac{\partial^2 \varepsilon_{xy}}{\partial x \partial z}$$

$$\frac{\partial^2 \varepsilon_{yy}}{\partial z \partial x} + \frac{\partial^2 \varepsilon_{zx}}{\partial y^2} = \frac{\partial^2 \varepsilon_{yx}}{\partial y \partial z} + \frac{\partial^2 \varepsilon_{yz}}{\partial y \partial x}$$

$$\frac{\partial^2 \varepsilon_{zz}}{\partial x \partial y} + \frac{\partial^2 \varepsilon_{xy}}{\partial z^2} = \frac{\partial^2 \varepsilon_{zy}}{\partial z \partial x} + \frac{\partial^2 \varepsilon_{zx}}{\partial z \partial y}$$

$$\frac{1}{r^2} \frac{\partial^2 \varepsilon_{rr}}{\partial \theta^2} + \frac{\partial^2 \varepsilon_{tt}}{\partial r^2} - \frac{2}{r} \frac{\partial^2 \varepsilon_{rt}}{\partial r \partial \theta} - \frac{1}{r} \frac{\partial \varepsilon_{rr}}{\partial r} + \frac{2}{r} \frac{\partial \varepsilon_{tt}}{\partial r} - \frac{2}{r^2} \frac{\partial \varepsilon_{rt}}{\partial \theta} = 0$$

# Planar deformation

$$u_1 = u_1(x_1, x_2) \quad ; \quad u_2 = u_2(x_1, x_2) \quad ; \quad u_3 = u_3(x_1, x_2, x_3)$$



# Plane strain

planar deformation

$$u_1 = u_1(x_1, x_2) ; u_2 = u_2(x_1, x_2) ; u_3 = u_3(x_1, x_2, x_3)$$

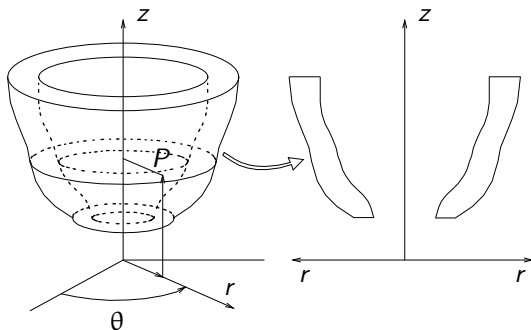
plane strain

$$u_1 = u_1(x_1, x_2) \quad ; \quad u_2 = u_2(x_1, x_2) \quad ; \quad u_3 = 0$$

$$\varepsilon_{33} = 0 \quad ; \quad \gamma_{13} = \gamma_{23} = 0$$

$$\text{compatibility} \quad : \quad \varepsilon_{11,22} + \varepsilon_{22,11} = 2\varepsilon_{12,12}$$

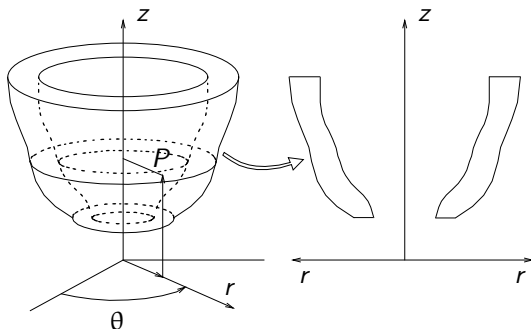
# Axi-symmetric deformation



$$\frac{\partial}{\partial \theta}(\quad) = 0 \quad \rightarrow \quad \vec{u} = u_r(r, z)\vec{e}_r(\theta) + u_t(r, z)\vec{e}_t(\theta) + u_z(r, z)\vec{e}_z$$

$$\underline{\varepsilon} = \frac{1}{2} \begin{bmatrix} 2u_{r,r} & -\frac{1}{r}(u_t) + u_{t,r} & u_{r,z} + u_{z,r} \\ -\frac{1}{r}(u_t) + u_{t,r} & 2\frac{1}{r}(u_r) & u_{t,z} \\ u_{z,r} + u_{r,z} & u_{t,z} & 2u_{z,z} \end{bmatrix}$$

# Axi-symmetric deformation with $u_t = 0$



$$\frac{\partial}{\partial \theta}(\quad) = 0 \text{ and } u_t = 0 \quad \rightarrow \quad \vec{u} = u_r(r, z)\vec{e}_r(\theta) + u_z(r, z)\vec{e}_z$$

$$\underline{\varepsilon} = \frac{1}{2} \begin{bmatrix} 2u_{r,r} & 0 & u_{r,z} + u_{z,r} \\ 0 & 2\frac{1}{r}(u_r) & 0 \\ u_{z,r} + u_{r,z} & 0 & 2u_{z,z} \end{bmatrix}$$

# Axi-symmetric plane strain

plane strain deformation

$$\left. \begin{array}{l} u_r = u_r(r, \theta) \\ u_t = u_t(r, \theta) \\ u_z = 0 \end{array} \right\} \rightarrow \varepsilon_{zz} = \gamma_{rz} = \gamma_{tz} = 0$$

linear strain matrix

$$\underline{\varepsilon} = \frac{1}{2} \begin{bmatrix} 2u_{r,r} & u_{t,r} - \frac{1}{r}(u_t) & 0 \\ u_{t,r} - \frac{1}{r}(u_t) & \frac{2}{r}(u_r) & 0 \\ 0 & 0 & 0 \end{bmatrix}$$

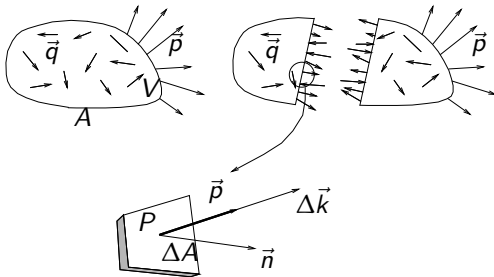
plane strain deformation with  $u_t = 0$

$$\left. \begin{array}{l} u_r = u_r(r) \\ u_z = 0 \end{array} \right\} \rightarrow \underline{\varepsilon} = \frac{1}{2} \begin{bmatrix} 2u_{r,r} & 0 & 0 \\ 0 & \frac{2}{r}(u_r) & 0 \\ 0 & 0 & 0 \end{bmatrix}$$

# STRESS

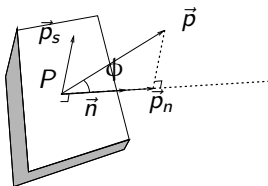
[back to index](#)

# Stress vector



$$\vec{p} = \lim_{\Delta A \rightarrow 0} \frac{\Delta \vec{k}}{\Delta A}$$

# Normal stress and shear stress



normal stress

tensile stress

compression stress

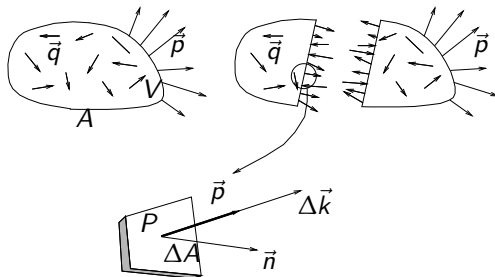
normal stress vector

shear stress vector

shear stress

:	$p_n = \vec{p} \cdot \vec{n}$
:	positive ( $\phi < \frac{\pi}{2}$ )
:	negative ( $\phi > \frac{\pi}{2}$ )
:	$\vec{p}_n = p_n \vec{n}$
:	$\vec{p}_s = \vec{p} - \vec{p}_n$
:	$p_s = \ \vec{p}_s\  = \sqrt{\ \vec{p}\ ^2 - p_n^2}$

# Cauchy stress tensor

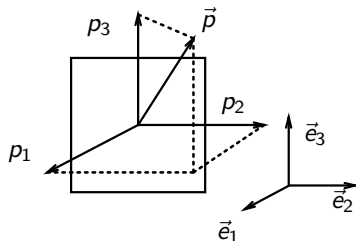


Theorem of Cauchy :

$$\exists! \text{ tensor } \sigma \text{ such that } : \quad \vec{p} = \sigma \cdot \vec{n}$$



# Cauchy stress matrix

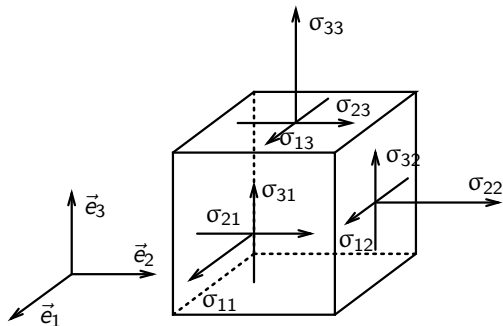


$$\vec{p} = \underline{\sigma} \cdot \vec{n} \rightarrow \tilde{\vec{e}}^T \underline{\tilde{p}} = \tilde{\vec{e}}^T \underline{\underline{\sigma}} \tilde{\vec{e}} \cdot \tilde{\vec{e}}^T \underline{\tilde{n}} = \tilde{\vec{e}}^T \underline{\underline{\sigma}} \underline{\tilde{n}}$$

$$\vec{n} = \vec{e}_1 \rightarrow$$

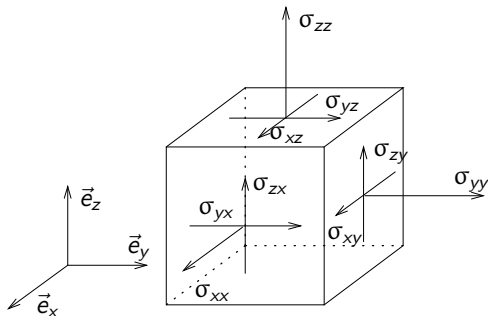
$$\begin{bmatrix} p_1 \\ p_2 \\ p_3 \end{bmatrix} = \begin{bmatrix} \sigma_{11} & \sigma_{12} & \sigma_{13} \\ \sigma_{21} & \sigma_{22} & \sigma_{23} \\ \sigma_{31} & \sigma_{32} & \sigma_{33} \end{bmatrix} \begin{bmatrix} 1 \\ 0 \\ 0 \end{bmatrix} = \begin{bmatrix} \sigma_{11} \\ \sigma_{21} \\ \sigma_{31} \end{bmatrix}$$

# Stress cube



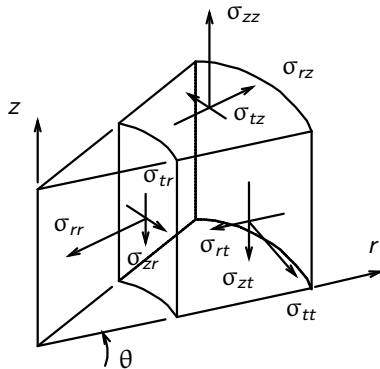
$$\underline{\sigma} = \begin{bmatrix} \sigma_{11} & \sigma_{12} & \sigma_{13} \\ \sigma_{21} & \sigma_{22} & \sigma_{23} \\ \sigma_{31} & \sigma_{32} & \sigma_{33} \end{bmatrix}$$

# Cartesian components



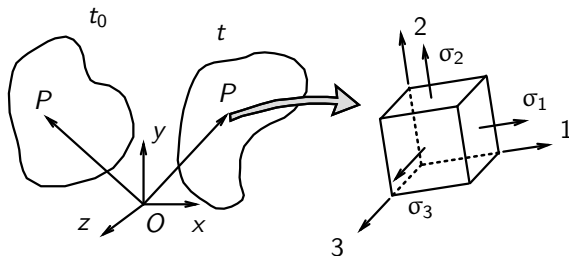
$$\underline{\sigma} = \begin{bmatrix} \sigma_{xx} & \sigma_{xy} & \sigma_{xz} \\ \sigma_{yx} & \sigma_{yy} & \sigma_{yz} \\ \sigma_{zx} & \sigma_{zy} & \sigma_{zz} \end{bmatrix}$$

# Cylindrical components



$$\underline{\underline{\sigma}} = \begin{bmatrix} \sigma_{rr} & \sigma_{r\theta} & \sigma_{rz} \\ \sigma_{\theta r} & \sigma_{\theta\theta} & \sigma_{\theta z} \\ \sigma_{zr} & \sigma_{z\theta} & \sigma_{zz} \end{bmatrix}$$

# Principal stresses and directions



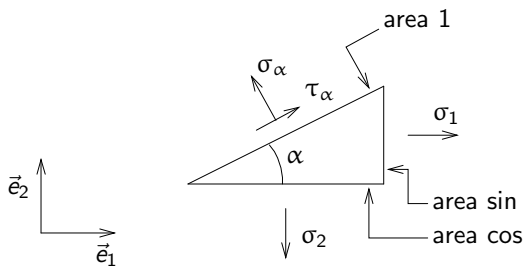
spectral form

$$\left. \begin{aligned} \boldsymbol{\sigma} \cdot \vec{n}_1 &= \sigma_1 \vec{n}_1 \\ \boldsymbol{\sigma} \cdot \vec{n}_2 &= \sigma_2 \vec{n}_2 \\ \boldsymbol{\sigma} \cdot \vec{n}_3 &= \sigma_3 \vec{n}_3 \end{aligned} \right\} \rightarrow \boldsymbol{\sigma} = \sigma_1 \vec{n}_1 \vec{n}_1 + \sigma_2 \vec{n}_2 \vec{n}_2 + \sigma_3 \vec{n}_3 \vec{n}_3$$

principal stress matrix

$$\underline{\underline{\sigma}}_P = \begin{bmatrix} \sigma_1 & 0 & 0 \\ 0 & \sigma_2 & 0 \\ 0 & 0 & \sigma_3 \end{bmatrix}$$

# Stress transformation



$$\boldsymbol{\sigma} = \sigma_1 \vec{e}_1 + \sigma_2 \vec{e}_2$$

$$\vec{n} = -\sin(\alpha) \vec{e}_1 + \cos(\alpha) \vec{e}_2$$

$$\vec{p} = \boldsymbol{\sigma} \cdot \vec{n} = -\sigma_1 \sin(\alpha) \vec{e}_1 + \sigma_2 \cos(\alpha) \vec{e}_2$$

$$\sigma_\alpha = \sigma_1 \sin^2(\alpha) + \sigma_2 \cos^2(\alpha)$$

$$\tau_\alpha = (\sigma_2 - \sigma_1) \sin(\alpha) \cos(\alpha)$$

# Mohr's circles of stress

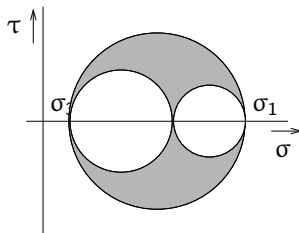
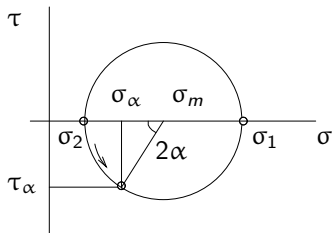
$$\begin{aligned}\sigma_\alpha &= \sigma_1 \sin^2(\alpha) + \sigma_2 \cos^2(\alpha) = \sigma_1 \left(\frac{1}{2} - \frac{1}{2} \cos(2\alpha)\right) + \sigma_2 \left(\frac{1}{2} + \frac{1}{2} \cos(2\alpha)\right) \\ &= \frac{1}{2}(\sigma_1 + \sigma_2) - \frac{1}{2}(\sigma_1 - \sigma_2) \cos(2\alpha) \rightarrow\end{aligned}$$

$$(1) \quad \left\{ \sigma_\alpha - \frac{1}{2}(\sigma_1 + \sigma_2) \right\}^2 = \left\{ \frac{1}{2}(\sigma_1 - \sigma_2) \right\}^2 \cos^2(2\alpha)$$

$$\tau_\alpha = -\cos(\alpha) \sin(\alpha) \sigma_1 + \cos(\alpha) \sin(\alpha) \sigma_2 = \frac{1}{2}(\sigma_2 - \sigma_1) \sin(2\alpha) \rightarrow$$

$$(2) \quad \tau_\alpha^2 = \left\{ \frac{1}{2}(\sigma_2 - \sigma_1) \right\}^2 \sin^2(2\alpha)$$

$$(1) + (2) \rightarrow \left\{ \sigma_\alpha - \frac{1}{2}(\sigma_1 + \sigma_2) \right\}^2 + \tau_\alpha^2 = \left\{ \frac{1}{2}(\sigma_1 - \sigma_2) \right\}^2$$



# Mohr's circles of stress

inside  $\sigma_1, \sigma_3$ -circle

$$\begin{aligned}\{\sigma - \tfrac{1}{2}(\sigma_1 + \sigma_3)\}^2 + \tau^2 &= \sigma^2 + \tau^2 = \|\vec{p}\|^2 = \vec{p} \cdot \vec{p} = \underline{n}^T \underline{\sigma}^T \underline{\sigma} \underline{n} \\ &= n_1^2 \alpha^2 + n_2^2 \beta^2 + n_3^2 \alpha^2\end{aligned}$$

with  $\beta^2 = (\sigma_2 - \tfrac{1}{2}(\sigma_1 + \sigma_3))^2 \leq \alpha^2 = (\sigma_1 - \tfrac{1}{2}(\sigma_1 + \sigma_3))^2 \rightarrow \sigma^2 + \tau^2 \leq \alpha^2$

outside  $\sigma_2, \sigma_3$ -circle

$$\begin{aligned}\{\sigma - \tfrac{1}{2}(\sigma_3 + \sigma_2)\}^2 + \tau^2 &= \sigma^2 + \tau^2 = \|\vec{p}\|^2 = \vec{p} \cdot \vec{p} = \underline{n}^T \underline{\sigma}^T \underline{\sigma} \underline{n} \\ &= n_1^2 \beta^2 + n_2^2 \alpha^2 + n_3^2 \alpha^2\end{aligned}$$

with  $\beta^2 = (\sigma_1 - \tfrac{1}{2}(\sigma_3 + \sigma_2))^2 \geq \alpha^2 = (\sigma_2 - \tfrac{1}{2}(\sigma_3 + \sigma_2))^2 \rightarrow \sigma^2 + \tau^2 \geq \alpha^2$

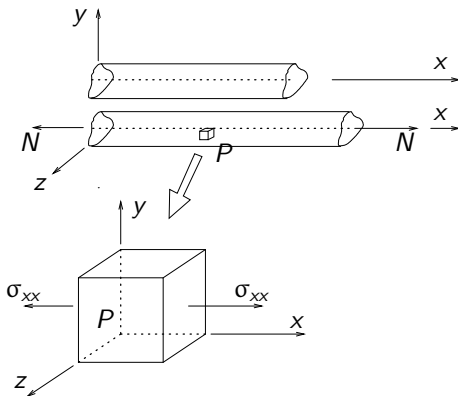
outside  $\sigma_1, \sigma_2$ -circle

$$\begin{aligned}\{\sigma - \tfrac{1}{2}(\sigma_1 + \sigma_2)\}^2 + \tau^2 &= \sigma^2 + \tau^2 = \|\vec{p}\|^2 = \vec{p} \cdot \vec{p} = \underline{n}^T \underline{\sigma}^T \underline{\sigma} \underline{n} \\ &= n_1^2 \alpha^2 + n_2^2 \alpha^2 + n_3^2 \beta^2\end{aligned}$$

with  $\beta^2 = (\sigma_3 - \tfrac{1}{2}(\sigma_1 + \sigma_2))^2 \geq \alpha^2 = (\sigma_2 - \tfrac{1}{2}(\sigma_1 + \sigma_2))^2 \rightarrow \sigma^2 + \tau^2 \geq \alpha^2$



# Uni-axial stress



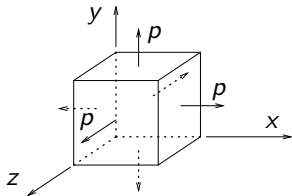
true or Cauchy stress

$$\sigma = \frac{N}{A} = \sigma_{xx} \quad \rightarrow \quad \boldsymbol{\sigma} = \sigma_{xx} \vec{e}_x \vec{e}_x$$

engineering stress

$$\sigma_n = \frac{N}{A_0}$$

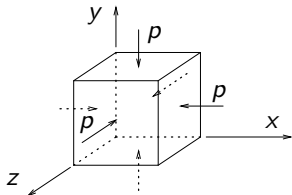
# Hydrostatic stress



$$\sigma_{xx} = p$$

$$\sigma_{yy} = p$$

$$\sigma_{zz} = p$$

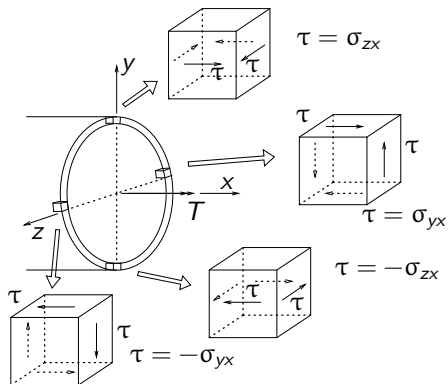


$$\sigma_{xx} = -p$$

$$\sigma_{yy} = -p$$

$$\sigma_{zz} = -p$$

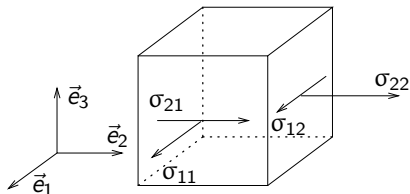
# Shear stress



$$\tau = \frac{T}{2\pi R^2 t}$$

$$\sigma = \tau(\vec{e}_i \vec{e}_j + \vec{e}_j \vec{e}_i) \text{ with } i \neq j$$

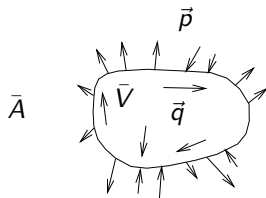
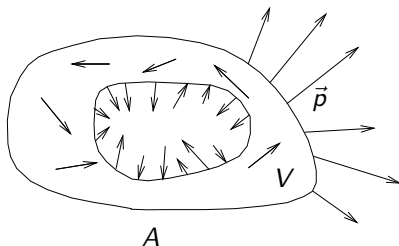
# Plane stress



$$\sigma_{33} = \sigma_{13} = \sigma_{23} = 0 \quad \rightarrow \quad \boldsymbol{\sigma} \cdot \vec{e}_3 = \vec{0} \quad \rightarrow$$

relevant stresses :  $\sigma_{11}, \sigma_{22}, \sigma_{12}$

# Resulting force on arbitrary material volume

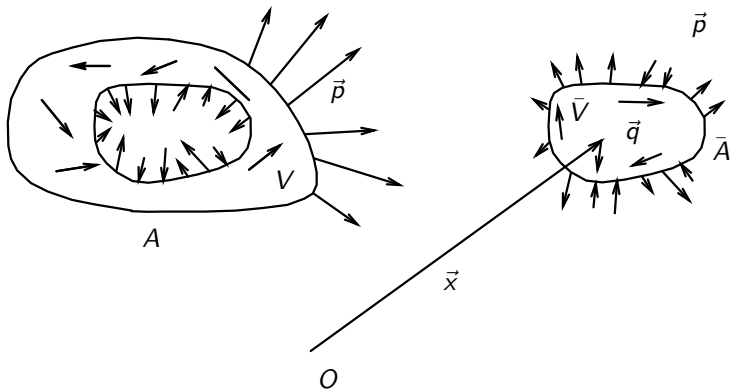


$$\vec{K} = \int_{\bar{V}} \rho \vec{q} dV + \int_{\bar{A}} \vec{p} dA = \int_{\bar{V}} \rho \vec{q} dV + \int_{\bar{A}} \vec{n} \cdot \vec{\sigma}^T dA$$

Gauss theorem :  $\int_{\bar{A}} \vec{n} \cdot ( ) dA = \int_{\bar{V}} \vec{\nabla} \cdot ( ) dV \rightarrow$

$$\vec{K} = \int_{\bar{V}} [\rho \vec{q} + \vec{\nabla} \cdot \vec{\sigma}^T] dV$$

# Resulting moment on arbitrary material volume



$$\vec{M}_O = \int_{\bar{V}} \vec{x} * \rho \vec{q} dV + \int_{\bar{A}} \vec{x} * \vec{p} dA$$

## Resulting moment on total body

$$\begin{aligned}\vec{M}_O &= \int_V \vec{x} * \rho \vec{q} dV + \int_A \vec{x} * \vec{p} dA \\&= \int_V (\vec{x}_R + \vec{r}) * \rho \vec{q} dV + \int_A (\vec{x}_R + \vec{r}) * \vec{p} dA \\&= \vec{x}_R * \int_V \rho \vec{q} dV + \vec{x}_R * \int_A \vec{p} dA + \int_V \vec{r} * \rho \vec{q} dV + \int_A \vec{r} * \vec{p} dA \\&= \vec{x}_R * \vec{K} + \vec{M}_R \\&= \vec{x}_M * \vec{K} + \vec{M}_M \quad \rightarrow\end{aligned}$$

$$\vec{M}_R = (\vec{x}_M - \vec{x}_R) * \vec{K} + \vec{M}_M = \vec{r}_M * \vec{K} + \vec{M}_M$$

## BALANCE LAWS

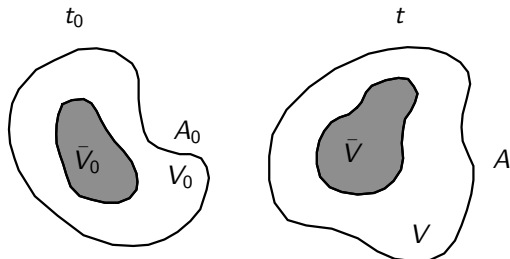
[back to index](#)



# Balance or conservation laws

- mass
- momentum
- moment of momentum
- energy

# Balance of mass

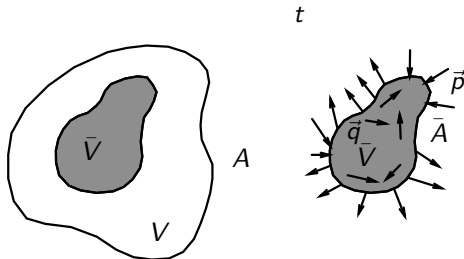


$$\int_{\bar{V}} \rho dV = \int_{\bar{V}_0} \rho_0 dV_0 \quad \forall \bar{V} \rightarrow \int_{\bar{V}_0} (\rho J - \rho_0) dV_0 = 0 \quad \forall \bar{V}_0 \rightarrow$$

$$\rho J = \rho_0 \quad \forall \vec{x} \in V(t)$$

$$dM = dM_0 \rightarrow \rho dV = \rho_0 dV_0 \rightarrow \rho J = \rho_0 \rightarrow \dot{\rho} J + \rho \dot{J} = 0$$

## Balance of momentum : global



$$\begin{aligned}
 \vec{K} &= \frac{D\vec{I}}{Dt} = \frac{D}{Dt} \int_{\bar{V}} \rho \vec{v} dV = \frac{D}{Dt} \int_{\bar{V}_0} \rho \vec{v} J dV_0 = \int_{\bar{V}_0} \frac{D}{Dt} (\rho \vec{v} J) dV_0 & \forall \quad \bar{V}_0 \\
 &= \int_{\bar{V}_0} \left( \dot{\rho} \vec{v} J + \rho \dot{\vec{v}} J + \rho \vec{v} \dot{J} \right) dV_0 & \forall \quad \bar{V}_0 \\
 &\quad \text{mass balance} \quad : \quad \dot{\rho} J + \rho \dot{J} = 0 \quad \rightarrow \\
 &= \int_{\bar{V}_0} \rho \dot{\vec{v}} J dV_0 = \int_{\bar{V}} \rho \dot{\vec{v}} dV & \forall \quad \bar{V}
 \end{aligned}$$

## Balance of momentum : local

$$\int_{\bar{V}} \left( \rho \vec{q} + \vec{\nabla} \cdot \boldsymbol{\sigma}^T \right) dV = \int_{\bar{V}} \rho \dot{\vec{v}} dV \quad \forall \quad \bar{V} \rightarrow$$

$$\vec{\nabla} \cdot \boldsymbol{\sigma}^T + \rho \vec{q} = \rho \dot{\vec{v}} = \rho \frac{\delta \vec{v}}{\delta t} + \rho \vec{v} \cdot \left( \vec{\nabla} \vec{v} \right) \quad \forall \quad \vec{x} \in V(t)$$

$$\text{stationary} \left( \frac{\delta \vec{v}}{\delta t} = 0 \right)$$

static : equilibrium equation

$$\vec{\nabla} \cdot \boldsymbol{\sigma}^T + \rho \vec{q} = \rho \vec{v} \cdot \left( \vec{\nabla} \vec{v} \right)$$

$$\vec{\nabla} \cdot \boldsymbol{\sigma}^T + \rho \vec{q} = \vec{0}$$

## Equilibrium equations : Cartesian components

$$\vec{\nabla} \cdot \boldsymbol{\sigma}^c + \rho \vec{q} = \vec{0}$$

$$\boldsymbol{\sigma} = \boldsymbol{\sigma}^c$$

$$\sigma_{xx,x} + \sigma_{xy,y} + \sigma_{xz,z} + \rho q_x = 0$$

$$\sigma_{yx,x} + \sigma_{yy,y} + \sigma_{yz,z} + \rho q_y = 0$$

$$\sigma_{zx,x} + \sigma_{zy,y} + \sigma_{zz,z} + \rho q_z = 0$$

## Equilibrium equations : cylindrical components

$$\vec{\nabla} \cdot \boldsymbol{\sigma}^c + \rho \vec{q} = \vec{0}$$

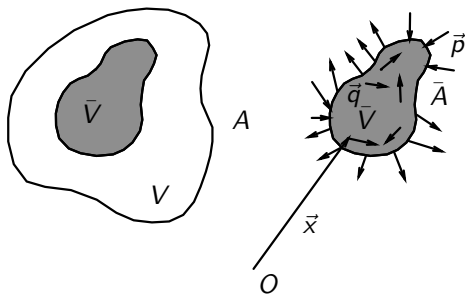
$$\boldsymbol{\sigma} = \boldsymbol{\sigma}^c$$

$$\sigma_{rr,r} + \frac{1}{r} \sigma_{rt,t} + \frac{1}{r} (\sigma_{rr} - \sigma_{tt}) + \sigma_{rz,z} + \rho q_r = 0$$

$$\sigma_{tr,r} + \frac{1}{r} \sigma_{tt,t} + \frac{1}{r} (\sigma_{tr} + \sigma_{rt}) + \sigma_{tz,z} + \rho q_t = 0$$

$$\sigma_{zr,r} + \frac{1}{r} \sigma_{zt,t} + \frac{1}{r} \sigma_{zr} + \sigma_{zz,z} + \rho q_z = 0$$

# Balance of moment of momentum : global



$$\begin{aligned}
 \vec{M}_O &= \frac{D\vec{L}_O}{Dt} = \frac{D}{Dt} \int_{\bar{V}} \vec{x} * \rho \vec{v} dV = \frac{D}{Dt} \int_{\bar{V}_0} \vec{x} * \rho \vec{v} J dV_0 = \int_{\bar{V}_0} \frac{D}{Dt} (\vec{x} * \rho \vec{v} J) dV_0 \\
 &= \int_{\bar{V}_0} \left( \dot{\vec{x}} * \rho \vec{v} J + \vec{x} * \dot{\rho} \vec{v} J + \vec{x} * \rho \dot{\vec{v}} J + \vec{x} * \rho \vec{v} \dot{J} \right) dV_0 \quad \forall \quad \bar{V}_0 \\
 &\quad \text{mass balance} \quad : \quad \dot{\rho} J + \rho \dot{J} = 0 \\
 &= \int_{\bar{V}_0} \vec{x} * \rho \dot{\vec{v}} J dV_0 = \int_{\bar{V}} \vec{x} * \rho \dot{\vec{v}} dV \quad \forall \quad \bar{V}
 \end{aligned}$$

## Balance of moment of momentum : local

$$\int_{\bar{V}} \vec{x} * \rho \vec{q} dV + \int_{\bar{A}} \vec{x} * \vec{p} dA = \int_{\bar{V}} \vec{x} * \rho \dot{\vec{v}} dV \quad \forall \quad \bar{V}$$

Transformation of surface integral with

$$\vec{x} * \vec{p} = {}^3\epsilon : (\vec{x} \vec{p})$$

$$\begin{aligned} \int_{\bar{A}} \vec{x} * \vec{p} dA &= \int_{\bar{A}} {}^3\epsilon : (\vec{x} \vec{p}) dA = \int_{\bar{A}} {}^3\epsilon : (\vec{x} \sigma \cdot \vec{n}) dA = \int_{\bar{A}} \vec{n} \cdot \{{}^3\epsilon : (\vec{x} \sigma)\}^c dA \\ &= \int_{\bar{V}} \vec{\nabla} \cdot \{{}^3\epsilon : (\vec{x} \sigma)\}^c dV \\ &= \int_{\bar{V}} \left[ (\vec{\nabla} \cdot \sigma^c) \vec{x} : {}^3\epsilon^c + \sigma \cdot (\vec{\nabla} \cdot \vec{x}) : {}^3\epsilon^c \right] dV \\ &= \int_{\bar{V}} \left[ (\vec{\nabla} \cdot \sigma^c) \vec{x} : {}^3\epsilon^c + \sigma : {}^3\epsilon^c \right] dV \\ &= \int_{\bar{V}} {}^3\epsilon : \sigma^c + \vec{x} * (\vec{\nabla} \cdot \sigma^c) dV \end{aligned}$$



## Balance of moment of momentum : local

$$\int_{\bar{V}} \vec{x} * \rho \vec{q} dV + \int_{\bar{V}} {}^3\epsilon : \sigma^c dV + \int_{\bar{V}} \vec{x} * (\vec{\nabla} \cdot \sigma^c) dV = \int_{\bar{V}} \vec{x} * \rho \dot{\vec{v}} dV \quad \forall \quad \bar{V} \rightarrow$$

$$\int_{\bar{V}} \vec{x} * \left[ \rho \vec{q} + (\vec{\nabla} \cdot \sigma^c) - \rho \dot{\vec{v}} \right] dV + \int_{\bar{V}} {}^3\epsilon : \sigma^c dV = \vec{0} \quad \forall \quad \bar{V} \rightarrow$$

$$\int_{\bar{V}} {}^3\epsilon : \sigma^c dV = \vec{0} \quad \forall \quad \bar{V} \rightarrow {}^3\epsilon : \sigma^c = \vec{0} \quad \forall \quad \vec{x} \in \bar{V}$$

$$\epsilon_{ijk} = -1|0|1 \rightarrow \begin{bmatrix} \sigma_{32} - \sigma_{23} \\ \sigma_{13} - \sigma_{31} \\ \sigma_{21} - \sigma_{12} \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \\ 0 \end{bmatrix} \rightarrow$$

$$\sigma^c = \sigma$$

$$\forall \quad \vec{x} \in V(t)$$

# Cartesian and cylindrical components

$$\underline{\sigma} = \underline{\sigma}^T \quad \rightarrow$$

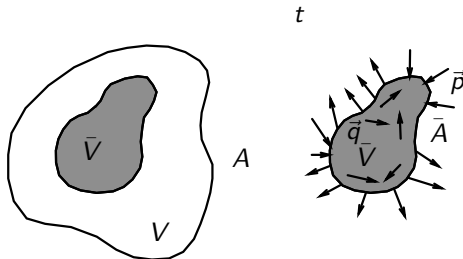
Cartesian	:	$\sigma_{xy} = \sigma_{yx}$	;	$\sigma_{yz} = \sigma_{zy}$	;	$\sigma_{zx} = \sigma_{xz}$
cylindrical	:	$\sigma_{rt} = \sigma_{tr}$	;	$\sigma_{tz} = \sigma_{zt}$	;	$\sigma_{zr} = \sigma_{rz}$

## Balance of energy

$$\frac{D}{Dt} (U_e + U_t) = \frac{D}{Dt} (U_k + U_i)$$

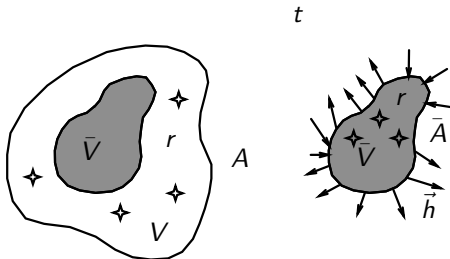
$U_e$	:	mechanical energy
$U_t$	:	thermal energy
$U_k$	:	kinetic energy
$U_i$	:	internal energy

# Mechanical energy



$$\begin{aligned}
 \dot{U}_e &= \int_{\bar{V}} \rho \vec{q} \cdot \vec{v} dV + \int_{\bar{A}} \vec{p} \cdot \vec{v} dA = \int_{\bar{V}} \{ \rho \vec{q} \cdot \vec{v} + \vec{\nabla} \cdot (\boldsymbol{\sigma}^c \cdot \vec{v}) \} dV \\
 \vec{\nabla} \cdot (\boldsymbol{\sigma}^c \cdot \vec{v}) &= (\vec{\nabla} \cdot \boldsymbol{\sigma}^c) \cdot \vec{v} + \boldsymbol{\sigma} : (\vec{\nabla} \vec{v}) \\
 &= \rho \dot{\vec{v}} \cdot \vec{v} - \rho \vec{q} \cdot \vec{v} + \boldsymbol{\sigma} : \mathbf{D} + \boldsymbol{\sigma} : \boldsymbol{\Omega} \\
 &= \int_{\bar{V}} (\rho \dot{\vec{v}} \cdot \vec{v} + \boldsymbol{\sigma} : \mathbf{D}) dV
 \end{aligned}$$

# Thermal energy



heat flux density

$$\vec{h} = \lim_{\Delta A \rightarrow 0} \frac{\vec{H}}{\Delta A} \quad [\text{J m}^{-2}]$$

$$\dot{U}_t = \int_{\bar{V}} \rho r \, dV - \int_{\bar{A}} \vec{n} \cdot \vec{h} \, dA = \int_{\bar{V}} (\rho r - \vec{\nabla} \cdot \vec{h}) \, dV$$

# Kinetic energy

$$U_k(t) = \int_{\vec{V}} \frac{1}{2} \rho \vec{v} \cdot \vec{v} dV$$

$$\begin{aligned}\dot{U}_k &= \frac{D}{Dt} \int_{\vec{V}} \frac{1}{2} \rho \vec{v} \cdot \vec{v} dV = \frac{D}{Dt} \int_{\vec{V}_0} \frac{1}{2} \rho \vec{v} \cdot \vec{v} J dV_0 \\&= \frac{1}{2} \int_{\vec{V}_0} \left\{ \dot{\rho} \vec{v} \cdot \vec{v} J + 2\rho \dot{\vec{v}} \cdot \vec{v} J + \rho \vec{v} \cdot \vec{v} \dot{J} \right\} dV_0 \\&= \int_{\vec{V}_0} \rho \dot{\vec{v}} \cdot \vec{v} J dV_0 = \int_{\vec{V}} \rho \dot{\vec{v}} \cdot \vec{v} dV\end{aligned}$$

# Internal energy

$$\begin{aligned}U_i(t) &= \int_{\bar{V}} \rho \phi \, dV \\ \dot{U}_i &= \frac{D}{Dt} \int_{\bar{V}} \rho \phi \, dV = \frac{D}{Dt} \int_{\bar{V}_0} \rho \phi J \, dV_0 \\ &= \int_{\bar{V}_0} \left\{ \dot{\rho} \phi J + \rho \dot{\phi} J + \rho \phi \dot{J} \right\} dV_0 \\ &= \int_{\bar{V}} \rho \dot{\phi} \, dV\end{aligned}$$

# Energy balance

$$\dot{U}_e + \dot{U}_t = \dot{U}_k + \dot{U}_i$$

$$\int_{\bar{V}} (\rho \dot{\vec{v}} \cdot \vec{v} + \boldsymbol{\sigma} : \mathbf{D} + \rho r - \vec{\nabla} \cdot \vec{h}) dV = \int_{\bar{V}} (\rho \dot{\vec{v}} \cdot \vec{v} + \rho \dot{\phi}) dV \quad \forall \quad \bar{V}$$

$$\int_{\bar{V}} \rho \dot{\phi} dV = \int_{\bar{V}} (\boldsymbol{\sigma} : \mathbf{D} + \rho r - \vec{\nabla} \cdot \vec{h}) dV \quad \forall \quad \bar{V}$$



# Energy equation

$$\rho \dot{\phi} = \boldsymbol{\sigma} : \mathbf{D} + \rho r - \vec{\nabla} \cdot \vec{h}$$

$$\dot{\phi} = C_p \dot{T} \quad (C_p : \text{specific heat})$$

$$\rho C_p \dot{T} = \boldsymbol{\sigma} : \mathbf{D} + \rho r - \vec{\nabla} \cdot \vec{h}$$

$$\vec{h} = -k \vec{\nabla} T \quad (k : \text{thermal conductivity})$$

$$\left. \begin{array}{l} \forall \quad \vec{x} \in V(t) \\ \\ \forall \quad \vec{x} \in V(t) \end{array} \right\} \rightarrow$$

$$\rho C_p \dot{T} - k \nabla^2 T = \boldsymbol{\sigma} : \mathbf{D} + \rho r$$

$$\forall \quad \vec{x} \in V(t)$$

# Mechanical power for three-dimensional deformation

$$\dot{W} = \boldsymbol{\sigma} : \mathbf{D} \quad \boldsymbol{\sigma} = \text{Cauchy stress tensor}$$

$$\begin{aligned} \dot{W}_0 &= [J\boldsymbol{\sigma}] : \mathbf{D} \\ &= \boldsymbol{\kappa} : \mathbf{D} \quad \boldsymbol{\kappa} = \text{Kirchhoff stress tensor} \end{aligned}$$

$$\begin{aligned} \dot{W}_0 &= J\boldsymbol{\sigma} : \mathbf{D} = J\boldsymbol{\sigma} : \frac{1}{2} \left( \dot{\mathbf{F}} \cdot \mathbf{F}^{-1} + (\dot{\mathbf{F}} \cdot \mathbf{F}^{-1})^c \right) = \\ &= J\boldsymbol{\sigma} : \left( \dot{\mathbf{F}} \cdot \mathbf{F}^{-1} \right) = J \left( \mathbf{F}^{-1} \cdot \boldsymbol{\sigma} \right) : \dot{\mathbf{F}} = \mathbf{S} : \dot{\mathbf{F}} = \mathbf{S} : \dot{\mathbf{U}} \\ &= \mathbf{S} : \dot{\boldsymbol{\varepsilon}} \quad \mathbf{S} = \text{1st-Piola-Kirchhoff stress tensor} \end{aligned}$$

$$\begin{aligned} \dot{W}_0 &= J\boldsymbol{\sigma} : \mathbf{D} = J\boldsymbol{\sigma} : \left( \mathbf{F}^{-c} \cdot \dot{\mathbf{E}} \cdot \mathbf{F}^{-1} \right) = J \left( \mathbf{F}^{-1} \cdot \boldsymbol{\sigma} \cdot \mathbf{F}^{-c} \right) : \dot{\mathbf{E}} \\ &= \mathbf{P} : \dot{\mathbf{E}} \quad \mathbf{P} = \text{2nd-Piola-Kirchhoff stress tensor} \end{aligned}$$

# Planar deformation

## Cartesian components

$$\sigma_{xx,x} + \sigma_{xy,y} + \rho q_x = 0$$

$$\sigma_{yx,x} + \sigma_{yy,y} + \rho q_y = 0$$

$$\sigma_{xy} = \sigma_{yx}$$

## cylindrical components

$$\sigma_{rr,r} + \frac{1}{r}\sigma_{rt,t} + \frac{1}{r}(\sigma_{rr} - \sigma_{tt}) + \rho q_r = 0$$

$$\sigma_{tr,r} + \frac{1}{r}\sigma_{tt,t} + \frac{1}{r}(\sigma_{tr} + \sigma_{rt}) + \rho q_t = 0$$

$$\sigma_{rt} = \sigma_{tr}$$

# Axisymmetric deformation

$$\sigma_{rr,r} + \frac{1}{r}(\sigma_{rr} - \sigma_{tt}) + \sigma_{rz,z} + \rho q_r = 0$$

$$\sigma_{tr,r} + \frac{2}{r}(\sigma_{tr}) + \sigma_{tz,z} + \rho q_t = 0 \quad (\text{if } u_t \neq 0)$$

$$\sigma_{zr,r} + \frac{1}{r}\sigma_{zr} + \sigma_{zz,z} + \rho q_z = 0$$

$$\sigma_{rt} = \sigma_{tr} \quad ; \quad \sigma_{tz} = \sigma_{zt} \quad (\text{if } u_t \neq 0)$$

$$\sigma_{zr} = \sigma_{rz}$$

planar

$$\sigma_{rr,r} + \frac{1}{r}(\sigma_{rr} - \sigma_{tt}) + \rho q_r = 0$$

$$\sigma_{tr,r} + \frac{2}{r}(\sigma_{tr}) + \rho q_t = 0 \quad (\text{if } u_t \neq 0)$$

$$\sigma_{rt} = \sigma_{tr} \quad (\text{if } u_t \neq 0)$$

# THREE-DIMENSIONAL MATERIAL MODELS

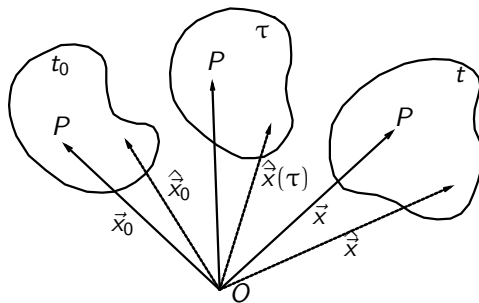
[back to index](#)

# Equations and unknowns

mass	$\rho J = \rho_0$
momentum	$\vec{\nabla} \cdot \boldsymbol{\sigma}^c + \rho \vec{q} = \rho \dot{\vec{v}}$
moment of momentum	$\boldsymbol{\sigma}^c = \boldsymbol{\sigma}$
density	$\rho$
position vector	$\vec{x}$
Cauchy stress tensor	$\boldsymbol{\sigma}$

$$\boldsymbol{\sigma} = \mathbf{N}(\vec{x})$$

# General constitutive equation

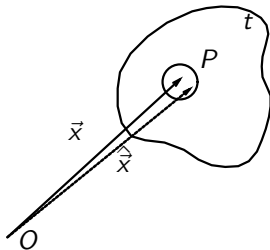


$$\sigma(\vec{x}, t) = \mathbf{N}[\hat{\vec{x}}, \tau \mid \forall \hat{\vec{x}} \in V; \forall \tau \leq t]$$

# Locality

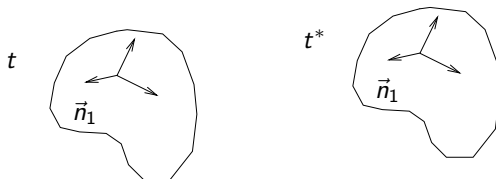
$$\left. \begin{aligned} \sigma(\vec{x}, t) &= \mathbf{N}[\hat{\vec{x}}, \tau \mid \forall \hat{\vec{x}} \in V ; \forall \tau \leq t] \\ \hat{\vec{x}} &= \vec{x} + d\vec{x} = \vec{x} + \mathbf{F}(\vec{x}) \cdot d\vec{x}_0 \end{aligned} \right\} \rightarrow$$

$$\sigma(\vec{x}, t) = \mathbf{N}(\vec{x}, \mathbf{F}(\vec{x}, \tau), \tau \mid \forall \tau \leq t)$$



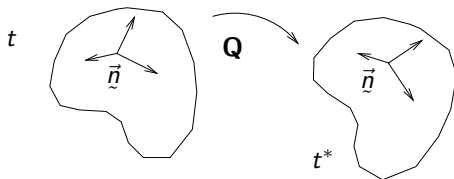


# Rigid body translation



$$\sigma(\vec{x}, t) = \mathbf{N}(\mathbf{F}(\vec{x}, \tau), \tau \mid \forall \tau \leq t)$$

# Rigid body rotation



$$\vec{n}_1^* = \mathbf{Q} \cdot \vec{n}_1$$

$$\vec{n}_2^* = \mathbf{Q} \cdot \vec{n}_2$$

$$\vec{n}_3^* = \mathbf{Q} \cdot \vec{n}_3$$

$$\boldsymbol{\sigma} = \sigma_1 \vec{n}_1 \vec{n}_1 + \sigma_2 \vec{n}_2 \vec{n}_2 + \sigma_3 \vec{n}_3 \vec{n}_3$$

$$\begin{aligned} \boldsymbol{\sigma}^* &= \sigma_1 \vec{n}_1^* \vec{n}_1^* + \sigma_2 \vec{n}_2^* \vec{n}_2^* + \sigma_3 \vec{n}_3^* \vec{n}_3^* \\ &= \sigma_1 \mathbf{Q} \cdot \vec{n}_1 \vec{n}_1 \cdot \mathbf{Q}^c + \sigma_2 \mathbf{Q} \cdot \vec{n}_2 \vec{n}_2 \cdot \mathbf{Q}^c + \sigma_3 \mathbf{Q} \cdot \vec{n}_3 \vec{n}_3 \cdot \mathbf{Q}^c \\ &= \mathbf{Q} \cdot [\sigma_1 \vec{n}_1 \vec{n}_1 + \sigma_2 \vec{n}_2 \vec{n}_2 + \sigma_3 \vec{n}_3 \vec{n}_3] \cdot \mathbf{Q}^c = \mathbf{Q} \cdot \boldsymbol{\sigma} \cdot \mathbf{Q}^c \end{aligned}$$

$$\mathbf{F} = \mathbf{R} \cdot \mathbf{U} \quad \rightarrow \quad \mathbf{F}^* = \mathbf{R}^* \cdot \mathbf{U} = \mathbf{Q} \cdot \mathbf{R} \cdot \mathbf{U} \quad \rightarrow \quad \mathbf{F}^* = \mathbf{Q} \cdot \mathbf{F}$$

objectivity requirement

$$\mathbf{Q}(t) \cdot \mathbf{N}(\mathbf{F}(\tau) \mid \forall \tau \leq t) \cdot \mathbf{Q}^c(t) = \mathbf{N}(\mathbf{Q} \cdot \mathbf{F}(\tau) \mid \forall \tau \leq t) \quad \forall \quad \mathbf{Q}$$

## Example

$$\sigma = C\mathbf{E} = C\frac{1}{2}(\mathbf{C} - \mathbf{I}) = C\frac{1}{2}(\mathbf{F}^T \cdot \mathbf{F} - \mathbf{I})$$

$$\sigma^* = \mathbf{Q} \cdot \sigma \cdot \mathbf{Q}^T$$

$$\mathbf{F}^* = \mathbf{Q} \cdot \mathbf{F}$$

$$\mathbf{E}^* = \frac{1}{2}(\mathbf{F}^T \cdot \mathbf{Q}^T \cdot \mathbf{Q} \cdot \mathbf{F} - \mathbf{I}) = \frac{1}{2}(\mathbf{F}^T \cdot \mathbf{F} - \mathbf{I}) = \mathbf{E}$$

$$\sigma^* = C\mathbf{E}$$

NOT OBJECTIVE

## Example

$$\sigma = C\mathbf{A} = C\frac{1}{2}(\mathbf{B} - \mathbf{I}) = C\frac{1}{2}(\mathbf{F} \cdot \mathbf{F}^T - \mathbf{I})$$

$$\sigma^* = \mathbf{Q} \cdot \sigma \cdot \mathbf{Q}^T$$

$$\mathbf{F}^* = \mathbf{Q} \cdot \mathbf{F}$$

$$\mathbf{A}^* = \frac{1}{2}(\mathbf{Q} \cdot \mathbf{F} \cdot \mathbf{F}^T \cdot \mathbf{Q}^T - \mathbf{I}) = \frac{1}{2}\mathbf{Q} \cdot (\mathbf{F} \cdot \mathbf{F}^T - \mathbf{I}) \cdot \mathbf{Q}^T = \mathbf{Q} \cdot \mathbf{A} \cdot \mathbf{Q}^c$$

$$\sigma^* = C\mathbf{A}^*$$

OBJECTIVE

## Example

$$\boldsymbol{\sigma} = -p\mathbf{I} + 2\eta\mathbf{D}$$

$$\mathbf{D} = \frac{1}{2}(\mathbf{L} + \mathbf{L}^c) \quad \text{with} \quad \mathbf{L} = \dot{\mathbf{F}} \cdot \mathbf{F}^{-1}$$

$$\boldsymbol{\sigma}^* = \mathbf{Q} \cdot \boldsymbol{\sigma} \cdot \mathbf{Q}^T$$

$$\mathbf{F}^* = \mathbf{Q} \cdot \mathbf{F} \quad ; \quad \mathbf{F}^{*-1} = \mathbf{F}^{-1} \cdot \mathbf{Q}^c \quad ; \quad \dot{\mathbf{F}}^* = \dot{\mathbf{Q}} \cdot \mathbf{F} + \mathbf{Q} \cdot \dot{\mathbf{F}}$$

$$\mathbf{L}^* = (\dot{\mathbf{Q}} \cdot \mathbf{F} + \mathbf{Q} \cdot \dot{\mathbf{F}}) \cdot \mathbf{F}^{-1} \cdot \mathbf{Q}^c = \dot{\mathbf{Q}} \cdot \mathbf{Q}^c + \mathbf{Q} \cdot \dot{\mathbf{F}} \cdot \mathbf{F}^{-1} \cdot \mathbf{Q}^c$$

$$\mathbf{D}^* = \frac{1}{2} \left[ \dot{\mathbf{Q}} \cdot \mathbf{Q}^c + \mathbf{Q} \cdot \dot{\mathbf{F}} \cdot \mathbf{F}^{-1} \cdot \mathbf{Q}^c + \mathbf{Q} \cdot \dot{\mathbf{Q}}^c + \mathbf{Q} \cdot (\dot{\mathbf{F}} \cdot \mathbf{F}^{-1})^c \cdot \mathbf{Q}^c \right]$$

$$\begin{aligned} \mathbf{Q} \cdot \mathbf{Q}^c &= \mathbf{I} \quad \rightarrow \quad \dot{\mathbf{Q}} \cdot \mathbf{Q}^c + \mathbf{Q} \cdot \dot{\mathbf{Q}}^c \\ &= \mathbf{Q} \cdot \mathbf{D} \cdot \mathbf{Q}^c \end{aligned}$$

$$\boldsymbol{\sigma}^* = -p\mathbf{I} + 2\eta\mathbf{D}^*$$

OBJECTIVE

# Special stress tensors

- choose invariant stress tensor
- choose invariant rate of stress tensor

# Invariant stress tensor

$$\mathbf{S} = \mathbf{A} \cdot \boldsymbol{\sigma} \cdot \mathbf{A}^c$$

$$\left. \begin{aligned} \mathbf{S}^* &= \mathbf{A}^* \cdot \boldsymbol{\sigma}^* \cdot \mathbf{A}^{*c} = \mathbf{A}^* \cdot \mathbf{Q} \cdot \boldsymbol{\sigma} \cdot \mathbf{Q}^c \cdot \mathbf{A}^{*c} \\ \text{define} \quad \mathbf{A}^* &= \mathbf{A} \cdot \mathbf{Q}^c \end{aligned} \right\} \rightarrow$$

$$\mathbf{S}^* = \mathbf{A} \cdot \mathbf{Q}^c \cdot \mathbf{Q} \cdot \boldsymbol{\sigma} \cdot \mathbf{Q}^c \cdot \mathbf{Q} \cdot \mathbf{A}^c = \mathbf{A} \cdot \boldsymbol{\sigma} \cdot \mathbf{A}^c = \mathbf{S}$$

$\mathbf{S}$  = invariant for rigid rotation

# Invariant rate of stress tensor

$$\dot{\mathbf{S}} = \dot{\mathbf{A}} \cdot \boldsymbol{\sigma} \cdot \mathbf{A}^c + \mathbf{A} \cdot \dot{\boldsymbol{\sigma}} \cdot \mathbf{A}^c + \mathbf{A} \cdot \boldsymbol{\sigma} \cdot \dot{\mathbf{A}}^c$$

$$\begin{aligned} \dot{\mathbf{S}}^* &= \dot{\mathbf{A}}^* \cdot \boldsymbol{\sigma}^* \cdot \mathbf{A}^{*c} + \mathbf{A}^* \cdot \dot{\boldsymbol{\sigma}}^* \cdot \mathbf{A}^{*c} + \mathbf{A}^* \cdot \boldsymbol{\sigma}^* \cdot \dot{\mathbf{A}}^{*c} \\ &= (\dot{\mathbf{A}} \cdot \mathbf{Q}^c + \mathbf{A} \cdot \dot{\mathbf{Q}}^c) \cdot \mathbf{Q} \cdot \boldsymbol{\sigma} \cdot \mathbf{Q}^c \cdot \mathbf{Q} \cdot \mathbf{A}^c + \\ &\quad \mathbf{A} \cdot \mathbf{Q}^c \cdot (\dot{\mathbf{Q}} \cdot \boldsymbol{\sigma} \cdot \mathbf{Q}^c + \mathbf{Q} \cdot \dot{\boldsymbol{\sigma}} \cdot \mathbf{Q}^c + \mathbf{Q} \cdot \boldsymbol{\sigma} \cdot \dot{\mathbf{Q}}^c) \cdot \mathbf{Q} \cdot \mathbf{A}^c + \\ &\quad \mathbf{A} \cdot \mathbf{Q}^c \cdot \mathbf{Q} \cdot \boldsymbol{\sigma} \cdot \mathbf{Q}^c \cdot (\mathbf{Q} \cdot \dot{\mathbf{A}}^c + \dot{\mathbf{Q}} \cdot \mathbf{A}^c) \\ &= \dot{\mathbf{A}} \cdot \boldsymbol{\sigma} \cdot \mathbf{A}^c + \mathbf{A} \cdot \dot{\mathbf{Q}}^c \cdot \mathbf{Q} \cdot \boldsymbol{\sigma} \cdot \mathbf{A}^c + \mathbf{A} \cdot \mathbf{Q}^c \cdot \dot{\mathbf{Q}} \cdot \boldsymbol{\sigma} \cdot \mathbf{A}^c + \\ &\quad \mathbf{A} \cdot \dot{\boldsymbol{\sigma}} \cdot \mathbf{A}^c + \mathbf{A} \cdot \boldsymbol{\sigma} \cdot \dot{\mathbf{Q}}^c \cdot \mathbf{Q} \cdot \mathbf{A}^c + \mathbf{A} \cdot \boldsymbol{\sigma} \cdot \dot{\mathbf{A}}^c + \\ &\quad \mathbf{A} \cdot \boldsymbol{\sigma} \cdot \mathbf{Q}^c \cdot \dot{\mathbf{Q}} \cdot \mathbf{A}^c \\ &= \dot{\mathbf{A}} \cdot \boldsymbol{\sigma} \cdot \mathbf{A}^c + \mathbf{A} \cdot \dot{\boldsymbol{\sigma}} \cdot \mathbf{A}^c + \mathbf{A} \cdot \boldsymbol{\sigma} \cdot \mathbf{A}^c + \\ &\quad \mathbf{A} \cdot \dot{\mathbf{Q}}^c \cdot \mathbf{Q} \cdot \boldsymbol{\sigma} \cdot \mathbf{A}^c + \mathbf{A} \cdot \mathbf{Q}^c \cdot \dot{\mathbf{Q}} \cdot \boldsymbol{\sigma} \cdot \mathbf{A}^c + \\ &\quad \mathbf{A} \cdot \boldsymbol{\sigma} \cdot \dot{\mathbf{Q}}^c \cdot \mathbf{Q} \cdot \mathbf{A}^c + \mathbf{A} \cdot \boldsymbol{\sigma} \cdot \mathbf{Q}^c \cdot \dot{\mathbf{Q}} \cdot \mathbf{A}^c \\ &= \dot{\mathbf{S}} \quad \rightarrow \quad \dot{\mathbf{S}} = \text{invariant for rigid rotation} \end{aligned}$$



# Rate of Cauchy stress tensor

$$\mathbf{S} = \mathbf{A} \cdot \boldsymbol{\sigma} \cdot \mathbf{A}^c$$

$$\begin{aligned}\dot{\mathbf{S}} &= \dot{\mathbf{A}} \cdot \boldsymbol{\sigma} \cdot \mathbf{A}^c + \mathbf{A} \cdot \dot{\boldsymbol{\sigma}} \cdot \mathbf{A}^c + \mathbf{A} \cdot \boldsymbol{\sigma} \cdot \dot{\mathbf{A}}^c \\ &= \mathbf{A} \cdot \left\{ (\mathbf{A}^{-1} \cdot \dot{\mathbf{A}}) \cdot \boldsymbol{\sigma} + \boldsymbol{\sigma} \cdot (\mathbf{A}^{-1} \cdot \dot{\mathbf{A}})^c + \dot{\boldsymbol{\sigma}} \right\} \cdot \mathbf{A}^c = \mathbf{A} \cdot \overset{\circ}{\boldsymbol{\sigma}} \cdot \mathbf{A}^c\end{aligned}$$

$$\overset{\circ}{\boldsymbol{\sigma}} = \dot{\boldsymbol{\sigma}} + (\mathbf{A}^{-1} \cdot \dot{\mathbf{A}}) \cdot \boldsymbol{\sigma} + \boldsymbol{\sigma} \cdot (\mathbf{A}^{-1} \cdot \dot{\mathbf{A}})^c$$

$$\begin{aligned}\overset{\circ}{\boldsymbol{\sigma}}^* &= \dot{\boldsymbol{\sigma}}^* + (\mathbf{A}^{-1*} \cdot \dot{\mathbf{A}}^*) \cdot \boldsymbol{\sigma}^* + \boldsymbol{\sigma}^* \cdot (\mathbf{A}^{-1*} \cdot \dot{\mathbf{A}}^*)^c \\ \mathbf{A}^* &= \mathbf{A} \cdot \mathbf{Q}^c \quad \rightarrow \quad \mathbf{A}^{*-1} = \mathbf{A}^{-1*} = \mathbf{Q} \cdot \mathbf{A}^{-1} \\ \dot{\mathbf{A}}^* &= \dot{\mathbf{A}} \cdot \mathbf{Q}^c + \mathbf{A} \cdot \dot{\mathbf{Q}}^c \\ \mathbf{A}^{-1*} \cdot \dot{\mathbf{A}}^* &= \mathbf{Q} \cdot \mathbf{A}^{-1} \cdot \dot{\mathbf{A}} \cdot \mathbf{Q}^c + \mathbf{Q} \cdot \dot{\mathbf{Q}}^c \\ &= \dot{\boldsymbol{\sigma}}^* + \mathbf{Q} \cdot \mathbf{A}^{-1} \cdot \dot{\mathbf{A}} \cdot \mathbf{Q}^c \cdot \boldsymbol{\sigma}^* + \mathbf{Q} \cdot \dot{\mathbf{Q}}^c \cdot \boldsymbol{\sigma}^* + \\ &\quad \boldsymbol{\sigma}^* \cdot \mathbf{Q} \cdot (\mathbf{A}^{-1} \cdot \dot{\mathbf{A}})^c \cdot \mathbf{Q}^c + \boldsymbol{\sigma}^* \cdot \dot{\mathbf{Q}} \cdot \mathbf{Q}^c \\ &= \mathbf{Q} \cdot \overset{\circ}{\boldsymbol{\sigma}} \cdot \mathbf{Q}^c \quad \rightarrow \quad \overset{\circ}{\boldsymbol{\sigma}} = \text{objective}\end{aligned}$$

# Objective rates and associated tensors

general tensor

$$\mathbf{S} = \boldsymbol{\sigma}_O = \mathbf{A} \cdot \boldsymbol{\sigma} \cdot \mathbf{A}^c$$

$$\dot{\mathbf{S}} = \dot{\boldsymbol{\sigma}}_O = \mathbf{A} \cdot \overset{\odot}{\boldsymbol{\sigma}}_O \cdot \mathbf{A}^c$$

general rate

$$\overset{\odot}{\boldsymbol{\sigma}}_O = \dot{\boldsymbol{\sigma}} + (\mathbf{A}^{-1} \cdot \dot{\mathbf{A}}) \cdot \boldsymbol{\sigma} + \boldsymbol{\sigma} \cdot (\mathbf{A}^{-1} \cdot \dot{\mathbf{A}})^c$$

Truesdell tensor

$$\boldsymbol{\sigma}_T = \mathbf{F}^{-1} \cdot \boldsymbol{\sigma} \cdot \mathbf{F}^{-c}$$

$$\dot{\boldsymbol{\sigma}}_T = \mathbf{F}^{-1} \cdot \overset{\odot}{\boldsymbol{\sigma}}_T \cdot \mathbf{F}^{-c}$$

Truesdell rate

$$\overset{\odot}{\boldsymbol{\sigma}}_T = \overset{\nabla}{\boldsymbol{\sigma}} = \dot{\boldsymbol{\sigma}} - \mathbf{L} \cdot \boldsymbol{\sigma} - \boldsymbol{\sigma} \cdot \mathbf{L}^c$$

Jaumann tensor

$$\boldsymbol{\sigma}_J = \mathbf{Q}^{-1} \cdot \boldsymbol{\sigma} \cdot \mathbf{Q}^{-c} \quad \text{with} \quad \dot{\mathbf{Q}} = \boldsymbol{\Omega} \cdot \mathbf{Q}$$

$$\dot{\boldsymbol{\sigma}}_J = \mathbf{Q}^{-1} \cdot \overset{\odot}{\boldsymbol{\sigma}}_J \cdot \mathbf{Q}^{-c}$$

Jaumann rate

$$\overset{\odot}{\boldsymbol{\sigma}}_J = \overset{\circ}{\boldsymbol{\sigma}} = \dot{\boldsymbol{\sigma}} - \boldsymbol{\Omega} \cdot \boldsymbol{\sigma} - \boldsymbol{\sigma} \cdot \boldsymbol{\Omega}^c$$

Cotter-Rivlin tensor

$$\boldsymbol{\sigma}_C = \mathbf{F}^c \cdot \boldsymbol{\sigma} \cdot \mathbf{F}$$

$$\dot{\boldsymbol{\sigma}}_C = \mathbf{F}^c \cdot \overset{\odot}{\boldsymbol{\sigma}}_C \cdot \mathbf{F}$$

Cotter-Rivlin rate

$$\overset{\odot}{\boldsymbol{\sigma}}_C = \overset{\triangle}{\boldsymbol{\sigma}} = \dot{\boldsymbol{\sigma}} + \mathbf{L}^c \cdot \boldsymbol{\sigma} + \boldsymbol{\sigma} \cdot \mathbf{L}$$

Dienes tensor

$$\boldsymbol{\sigma}_D = \mathbf{R}^c \cdot \boldsymbol{\sigma} \cdot \mathbf{R} \quad \text{with} \quad \mathbf{F} = \mathbf{R} \cdot \mathbf{U}$$

$$\dot{\boldsymbol{\sigma}}_D = \mathbf{R}^c \cdot \overset{\odot}{\boldsymbol{\sigma}}_D \cdot \mathbf{R}$$

Dienes rate

$$\overset{\odot}{\boldsymbol{\sigma}}_D = \overset{\diamond}{\boldsymbol{\sigma}} = \dot{\boldsymbol{\sigma}} - (\dot{\mathbf{R}} \cdot \mathbf{R}^c) \cdot \boldsymbol{\sigma} - \boldsymbol{\sigma} \cdot (\dot{\mathbf{R}} \cdot \mathbf{R}^c)^c$$

# LINEAR ELASTIC MATERIAL

[back to index](#)

# Linear elastic material

tensor notation

$$\sigma = {}^4\mathbf{C} : \varepsilon$$

index notation

$$\sigma_{ij} = C_{ijkl} \varepsilon_{kl} \quad ; \quad i, j, k, l \in \{1, 2, 3\}$$

matrix notation

$$\underline{\underline{\sigma}} = \underline{\underline{C}} \underline{\underline{\varepsilon}}$$

$$\begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{33} \\ \sigma_{12} \\ \sigma_{21} \\ \sigma_{23} \\ \sigma_{32} \\ \sigma_{31} \\ \sigma_{13} \end{bmatrix} = \begin{bmatrix} C_{1111} & C_{1122} & C_{1133} & C_{1121} & C_{1112} & C_{1132} & C_{1123} & C_{1113} & C_{1131} \\ C_{2211} & C_{2222} & C_{2233} & C_{2221} & C_{2212} & C_{2232} & C_{2223} & C_{2213} & C_{2231} \\ C_{3311} & C_{3322} & C_{3333} & C_{3321} & C_{3312} & C_{3332} & C_{3323} & C_{3313} & C_{3331} \\ C_{1211} & C_{1222} & C_{1233} & C_{1221} & C_{1212} & C_{1232} & C_{1223} & C_{1213} & C_{1231} \\ C_{2111} & C_{2122} & C_{2133} & C_{2121} & C_{2112} & C_{2132} & C_{2123} & C_{2113} & C_{2131} \\ C_{2311} & C_{2322} & C_{2333} & C_{2321} & C_{2312} & C_{2332} & C_{2323} & C_{2313} & C_{2331} \\ C_{3211} & C_{3222} & C_{3233} & C_{3221} & C_{3212} & C_{3232} & C_{3223} & C_{3213} & C_{3231} \\ C_{3111} & C_{3122} & C_{3133} & C_{3121} & C_{3112} & C_{3132} & C_{3123} & C_{3113} & C_{3131} \\ C_{1311} & C_{1322} & C_{1333} & C_{1321} & C_{1312} & C_{1332} & C_{1323} & C_{1313} & C_{1331} \end{bmatrix} \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \varepsilon_{33} \\ \varepsilon_{12} \\ \varepsilon_{21} \\ \varepsilon_{23} \\ \varepsilon_{32} \\ \varepsilon_{31} \\ \varepsilon_{13} \end{bmatrix}$$

# Symmetry

$$\begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{33} \\ \sigma_{12} \\ \sigma_{21} \\ \sigma_{23} \\ \sigma_{32} \\ \sigma_{31} \\ \sigma_{13} \end{bmatrix} = \begin{bmatrix} C_{1111} & C_{1122} & C_{1133} & C_{1121} & C_{1112} & C_{1132} & C_{1123} & C_{1113} & C_{1131} \\ C_{2211} & C_{2222} & C_{2233} & C_{2221} & C_{2212} & C_{2232} & C_{2223} & C_{2213} & C_{2231} \\ C_{3311} & C_{3322} & C_{3333} & C_{3321} & C_{3312} & C_{3332} & C_{3323} & C_{3313} & C_{3331} \\ C_{1211} & C_{1222} & C_{1233} & C_{1221} & C_{1212} & C_{1232} & C_{1223} & C_{1213} & C_{1231} \\ C_{2111} & C_{2122} & C_{2133} & C_{2121} & C_{2112} & C_{2132} & C_{2123} & C_{2113} & C_{2131} \\ C_{2311} & C_{2322} & C_{2333} & C_{2321} & C_{2312} & C_{2332} & C_{2323} & C_{2313} & C_{2331} \\ C_{3211} & C_{3222} & C_{3233} & C_{3221} & C_{3212} & C_{3232} & C_{3223} & C_{3213} & C_{3231} \\ C_{3111} & C_{3122} & C_{3133} & C_{3121} & C_{3112} & C_{3132} & C_{3123} & C_{3113} & C_{3131} \\ C_{1311} & C_{1322} & C_{1333} & C_{1321} & C_{1312} & C_{1332} & C_{1323} & C_{1313} & C_{1331} \end{bmatrix} \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \varepsilon_{33} \\ \varepsilon_{12} \\ \varepsilon_{21} \\ \varepsilon_{23} \\ \varepsilon_{32} \\ \varepsilon_{31} \\ \varepsilon_{13} \end{bmatrix}$$

specific energy  $W = \frac{1}{2} \underline{\underline{\varepsilon}}^T \underline{\underline{C}} \underline{\underline{\varepsilon}} \rightarrow$

symmetry

$$\underline{\underline{C}} = \underline{\underline{C}}^T$$

# Symmetric stresses

$$\begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{33} \\ \sigma_{12} \\ \sigma_{21} \\ \sigma_{23} \\ \sigma_{32} \\ \sigma_{31} \\ \sigma_{13} \end{bmatrix} = \begin{bmatrix} C_{1111} & C_{1122} & C_{1133} & C_{1121} & C_{1112} & C_{1132} & C_{1123} & C_{1113} & C_{1131} \\ C_{2211} & C_{2222} & C_{2233} & C_{2221} & C_{2212} & C_{2232} & C_{2223} & C_{2213} & C_{2231} \\ C_{3311} & C_{3322} & C_{3333} & C_{3321} & C_{3312} & C_{3332} & C_{3323} & C_{3313} & C_{3331} \\ C_{1211} & C_{1222} & C_{1233} & C_{1221} & C_{1212} & C_{1232} & C_{1223} & C_{1213} & C_{1231} \\ C_{2111} & C_{2122} & C_{2133} & C_{2121} & C_{2112} & C_{2132} & C_{2123} & C_{2113} & C_{2131} \\ C_{2311} & C_{2322} & C_{2333} & C_{2321} & C_{2312} & C_{2332} & C_{2323} & C_{2313} & C_{2331} \\ C_{3211} & C_{3222} & C_{3233} & C_{3221} & C_{3212} & C_{3232} & C_{3223} & C_{3213} & C_{3231} \\ C_{3111} & C_{3122} & C_{3133} & C_{3121} & C_{3112} & C_{3132} & C_{3123} & C_{3113} & C_{3131} \\ C_{1311} & C_{1322} & C_{1333} & C_{1321} & C_{1312} & C_{1332} & C_{1323} & C_{1313} & C_{1331} \end{bmatrix} \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \varepsilon_{33} \\ \varepsilon_{12} \\ \varepsilon_{21} \\ \varepsilon_{23} \\ \varepsilon_{32} \\ \varepsilon_{31} \\ \varepsilon_{13} \end{bmatrix}$$

$$\sigma_{ij} = \sigma_{ji}$$

$$\begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{33} \\ \sigma_{12} \\ \sigma_{23} \\ \sigma_{31} \end{bmatrix} = \begin{bmatrix} C_{1111} & C_{1122} & C_{1133} & C_{1121} & C_{1112} & C_{1132} & C_{1123} & C_{1113} & C_{1131} \\ C_{2211} & C_{2222} & C_{2233} & C_{2221} & C_{2212} & C_{2232} & C_{2223} & C_{2213} & C_{2231} \\ C_{3311} & C_{3322} & C_{3333} & C_{3321} & C_{3312} & C_{3332} & C_{3323} & C_{3313} & C_{3331} \\ C_{1211} & C_{1222} & C_{1233} & C_{1221} & C_{1212} & C_{1232} & C_{1223} & C_{1213} & C_{1231} \\ C_{2311} & C_{2322} & C_{2333} & C_{2321} & C_{2312} & C_{2332} & C_{2323} & C_{2313} & C_{2331} \\ C_{3111} & C_{3122} & C_{3133} & C_{3121} & C_{3112} & C_{3132} & C_{3123} & C_{3113} & C_{3131} \end{bmatrix} \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \varepsilon_{33} \\ \varepsilon_{12} \\ \varepsilon_{21} \\ \varepsilon_{23} \\ \varepsilon_{32} \\ \varepsilon_{31} \\ \varepsilon_{13} \end{bmatrix}$$

# Symmetric strains

$$\begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{33} \\ \sigma_{12} \\ \sigma_{23} \\ \sigma_{31} \end{bmatrix} = \begin{bmatrix} C_{1111} & C_{1122} & C_{1133} & C_{1121} & C_{1112} & C_{1132} & C_{1123} & C_{1113} & C_{1131} \\ C_{2211} & C_{2222} & C_{2233} & C_{2221} & C_{2212} & C_{2232} & C_{2223} & C_{2213} & C_{2231} \\ C_{3311} & C_{3322} & C_{3333} & C_{3321} & C_{3312} & C_{3332} & C_{3323} & C_{3313} & C_{3331} \\ C_{1211} & C_{1222} & C_{1233} & C_{1221} & C_{1212} & C_{1232} & C_{1223} & C_{1213} & C_{1231} \\ C_{2311} & C_{2322} & C_{2333} & C_{2321} & C_{2312} & C_{2332} & C_{2323} & C_{2313} & C_{2331} \\ C_{3111} & C_{3122} & C_{3133} & C_{3121} & C_{3112} & C_{3132} & C_{3123} & C_{3113} & C_{3131} \end{bmatrix} \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \varepsilon_{33} \\ \varepsilon_{12} \\ \varepsilon_{21} \\ \varepsilon_{23} \\ \varepsilon_{32} \\ \varepsilon_{31} \\ \varepsilon_{13} \end{bmatrix}$$

$$\varepsilon_{ij} = \varepsilon_{ji}$$

$$\begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{33} \\ \sigma_{12} \\ \sigma_{23} \\ \sigma_{31} \end{bmatrix} = \begin{bmatrix} C_{1111} & C_{1122} & C_{1133} & [C_{1121} + C_{1112}] & [C_{1132} + C_{1123}] & [C_{1113} + C_{1131}] \\ C_{2211} & C_{2222} & C_{2233} & [C_{2221} + C_{2212}] & [C_{2232} + C_{2223}] & [C_{2213} + C_{2231}] \\ C_{3311} & C_{3322} & C_{3333} & [C_{3321} + C_{3312}] & [C_{3332} + C_{3323}] & [C_{3313} + C_{3331}] \\ C_{1211} & C_{1222} & C_{1233} & [C_{1221} + C_{1212}] & [C_{1232} + C_{1223}] & [C_{1213} + C_{1231}] \\ C_{2311} & C_{2322} & C_{2333} & [C_{2321} + C_{2312}] & [C_{2332} + C_{2323}] & [C_{2313} + C_{2331}] \\ C_{3111} & C_{3122} & C_{3133} & [C_{3121} + C_{3112}] & [C_{3132} + C_{3123}] & [C_{3113} + C_{3131}] \end{bmatrix} \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \varepsilon_{33} \\ \varepsilon_{12} \\ \varepsilon_{23} \\ \varepsilon_{31} \end{bmatrix}$$

# Symmetric material parameters

$$\begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{33} \\ \sigma_{12} \\ \sigma_{23} \\ \sigma_{31} \end{bmatrix} = \begin{bmatrix} C_{1111} & C_{1122} & C_{1133} & [C_{1121} + C_{1112}] & [C_{1132} + C_{1123}] & [C_{1113} + C_{1131}] \\ C_{2211} & C_{2222} & C_{2233} & [C_{2221} + C_{2212}] & [C_{2232} + C_{2223}] & [C_{2213} + C_{2231}] \\ C_{3311} & C_{3322} & C_{3333} & [C_{3321} + C_{3312}] & [C_{3332} + C_{3323}] & [C_{3313} + C_{3331}] \\ C_{1211} & C_{1222} & C_{1233} & [C_{1221} + C_{1212}] & [C_{1232} + C_{1223}] & [C_{1213} + C_{1231}] \\ C_{2311} & C_{2322} & C_{2333} & [C_{2321} + C_{2312}] & [C_{2332} + C_{2323}] & [C_{2313} + C_{2331}] \\ C_{3111} & C_{3122} & C_{3133} & [C_{3121} + C_{3112}] & [C_{3132} + C_{3123}] & [C_{3113} + C_{3131}] \end{bmatrix} \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \varepsilon_{33} \\ \varepsilon_{12} \\ \varepsilon_{23} \\ \varepsilon_{31} \end{bmatrix}$$

$$C_{ijkl} = C_{ijlk}$$

$$\begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{33} \\ \sigma_{12} \\ \sigma_{23} \\ \sigma_{31} \end{bmatrix} = \begin{bmatrix} C_{1111} & C_{1122} & C_{1133} & 2C_{1121} & 2C_{1132} & 2C_{1113} \\ C_{2211} & C_{2222} & C_{2233} & 2C_{2221} & 2C_{2232} & 2C_{2213} \\ C_{3311} & C_{3322} & C_{3333} & 2C_{3321} & 2C_{3332} & 2C_{3313} \\ C_{1211} & C_{1222} & C_{1233} & 2C_{1221} & 2C_{1232} & 2C_{1213} \\ C_{2311} & C_{2322} & C_{2333} & 2C_{2321} & 2C_{2332} & 2C_{2313} \\ C_{3111} & C_{3122} & C_{3133} & 2C_{3121} & 2C_{3132} & 2C_{3113} \end{bmatrix} \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \varepsilon_{33} \\ \varepsilon_{12} \\ \varepsilon_{23} \\ \varepsilon_{31} \end{bmatrix}$$



# Shear strain

$$\begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{33} \\ \sigma_{12} \\ \sigma_{23} \\ \sigma_{31} \end{bmatrix} = \begin{bmatrix} C_{1111} & C_{1122} & C_{1133} & 2C_{1121} & 2C_{1132} & 2C_{1113} \\ C_{2211} & C_{2222} & C_{2233} & 2C_{2221} & 2C_{2232} & 2C_{2213} \\ C_{3311} & C_{3322} & C_{3333} & 2C_{3321} & 2C_{3332} & 2C_{3313} \\ C_{1211} & C_{1222} & C_{1233} & 2C_{1221} & 2C_{1232} & 2C_{1213} \\ C_{2311} & C_{2322} & C_{2333} & 2C_{2321} & 2C_{2332} & 2C_{2313} \\ C_{3111} & C_{3122} & C_{3133} & 2C_{3121} & 2C_{3132} & 2C_{3113} \end{bmatrix} \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \varepsilon_{33} \\ \varepsilon_{12} \\ \varepsilon_{23} \\ \varepsilon_{31} \end{bmatrix}$$

$$2\varepsilon_{ij} = \gamma_{ij}$$

$$\begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{33} \\ \sigma_{12} \\ \sigma_{23} \\ \sigma_{31} \end{bmatrix} = \begin{bmatrix} C_{1111} & C_{1122} & C_{1133} & C_{1121} & C_{1132} & C_{1113} \\ C_{2211} & C_{2222} & C_{2233} & C_{2221} & C_{2232} & C_{2213} \\ C_{3311} & C_{3322} & C_{3333} & C_{3321} & C_{3332} & C_{3313} \\ C_{1211} & C_{1222} & C_{1233} & C_{1221} & C_{1232} & C_{1213} \\ C_{2311} & C_{2322} & C_{2333} & C_{2321} & C_{2332} & C_{2313} \\ C_{3111} & C_{3122} & C_{3133} & C_{3121} & C_{3132} & C_{3113} \end{bmatrix} \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \varepsilon_{33} \\ \gamma_{12} \\ \gamma_{23} \\ \gamma_{31} \end{bmatrix}$$

# Material symmetry

monoclinic  $\rightarrow$  orthotropic  $\rightarrow$  quadratic  $\rightarrow$  transversal isotropic  $\rightarrow$  cubic  $\rightarrow$   
isotropic

# MATERIAL SYMMETRY

[back to index](#)

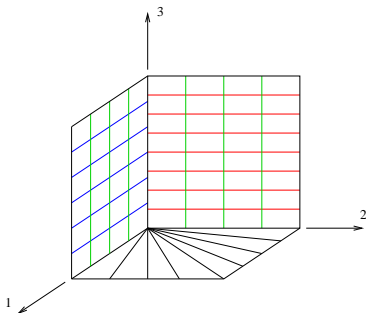
## Triclinic : no symmetry

$$\begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{33} \\ \sigma_{12} \\ \sigma_{23} \\ \sigma_{31} \end{bmatrix} = \begin{bmatrix} C_{1111} & C_{1122} & C_{1133} & C_{1121} & C_{1132} & C_{1113} \\ C_{2211} & C_{2222} & C_{2233} & C_{2221} & C_{2232} & C_{2213} \\ C_{3311} & C_{3322} & C_{3333} & C_{3321} & C_{3332} & C_{3313} \\ C_{1211} & C_{1222} & C_{1233} & C_{1221} & C_{1232} & C_{1213} \\ C_{2311} & C_{2322} & C_{2333} & C_{2321} & C_{2332} & C_{2313} \\ C_{3111} & C_{3122} & C_{3133} & C_{3121} & C_{3132} & C_{3113} \end{bmatrix} \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \varepsilon_{33} \\ \gamma_{12} \\ \gamma_{23} \\ \gamma_{31} \end{bmatrix}$$

21 material parameters

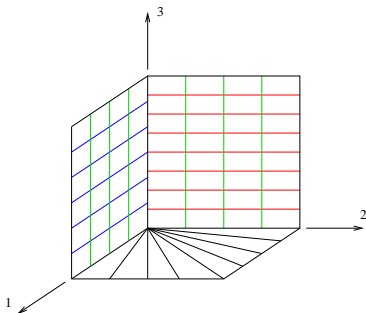
# Monoclinic : 1 symmetry plane (here 12)

$$\begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{33} \\ \sigma_{12} \\ \sigma_{23} \\ \sigma_{31} \end{bmatrix} = \begin{bmatrix} C_{1111} & C_{1122} & C_{1133} & C_{1121} & C_{1132} & C_{1113} \\ C_{2211} & C_{2222} & C_{2233} & C_{2221} & C_{2232} & C_{2213} \\ C_{3311} & C_{3322} & C_{3333} & C_{3321} & C_{3332} & C_{3313} \\ C_{1211} & C_{1222} & C_{1233} & C_{1221} & C_{1232} & C_{1213} \\ C_{2311} & C_{2322} & C_{2333} & C_{2321} & C_{2332} & C_{2313} \\ C_{3111} & C_{3122} & C_{3133} & C_{3121} & C_{3132} & C_{3113} \end{bmatrix} \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \varepsilon_{33} \\ \gamma_{12} \\ \gamma_{23} \\ \gamma_{31} \end{bmatrix}$$



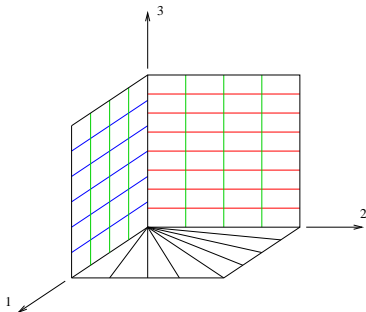
# Monoclinic : tensile test

$$\begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{33} \\ \sigma_{12} \\ \sigma_{23} \\ \sigma_{31} \end{bmatrix} = \begin{bmatrix} C_{1111} & C_{1122} & C_{1133} & C_{1121} & C_{1132} & C_{1113} \\ C_{2211} & C_{2222} & C_{2233} & C_{2221} & C_{2232} & C_{2213} \\ C_{3311} & C_{3322} & C_{3333} & C_{3321} & C_{3332} & C_{3313} \\ C_{1211} & C_{1222} & C_{1233} & C_{1221} & C_{1232} & C_{1213} \\ C_{2311} & C_{2322} & C_{2333} & C_{2321} & C_{2332} & C_{2313} \\ C_{3111} & C_{3122} & C_{3133} & C_{3121} & C_{3132} & C_{3113} \end{bmatrix} \begin{bmatrix} \varepsilon_{11} \\ 0 \\ 0 \\ 0 \\ 0 \\ 0 \end{bmatrix}$$

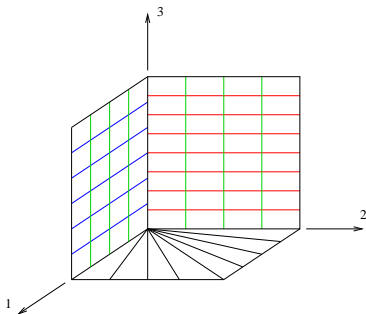


# Monoclinic : tensile test

$$\begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{33} \\ \sigma_{12} \\ \sigma_{23} \\ \sigma_{31} \end{bmatrix} = \begin{bmatrix} C_{1111} & C_{1122} & C_{1133} & C_{1121} & C_{1132} & C_{1113} \\ \textcolor{green}{C}_{2211} & C_{2222} & C_{2233} & C_{2221} & C_{2232} & C_{2213} \\ \textcolor{green}{C}_{3311} & \textcolor{green}{C}_{3322} & C_{3333} & C_{3321} & C_{3332} & C_{3313} \\ \textcolor{green}{C}_{1211} & \textcolor{green}{C}_{1222} & \textcolor{green}{C}_{1233} & C_{1221} & C_{1232} & C_{1213} \\ 0 & \textcolor{green}{C}_{2322} & \textcolor{green}{C}_{2333} & \textcolor{green}{C}_{2321} & C_{2332} & C_{2313} \\ 0 & \textcolor{green}{C}_{3122} & \textcolor{green}{C}_{3133} & \textcolor{green}{C}_{3121} & \textcolor{green}{C}_{3132} & C_{3113} \end{bmatrix} \begin{bmatrix} \varepsilon_{11} \\ 0 \\ 0 \\ 0 \\ 0 \\ 0 \end{bmatrix}$$



# Monoclinic : 1 symmetry plane (here 12)

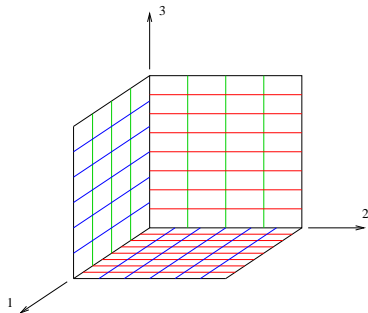


$$\begin{bmatrix}
 C_{1111} & C_{1122} & C_{1133} & C_{1121} & 0 & 0 \\
 C_{2211} & C_{2222} & C_{2233} & C_{2221} & 0 & 0 \\
 C_{3311} & C_{3322} & C_{3333} & C_{3321} & 0 & 0 \\
 C_{1211} & C_{1222} & C_{1233} & C_{1221} & 0 & 0 \\
 0 & 0 & 0 & 0 & C_{2332} & C_{2313} \\
 0 & 0 & 0 & 0 & C_{3132} & C_{3113}
 \end{bmatrix}$$

13 material parameters



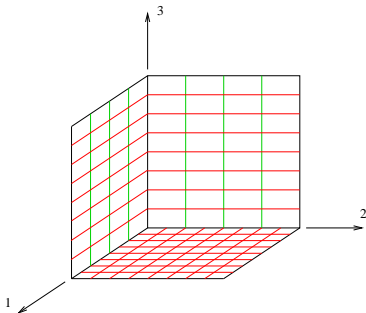
## Orthotropic : 3 symmetry planes (12, 23, 31)



$$\underline{\underline{C}} = \begin{bmatrix} A & Q & R & 0 & 0 & 0 \\ Q & B & S & 0 & 0 & 0 \\ R & S & C & 0 & 0 & 0 \\ 0 & 0 & 0 & K & 0 & 0 \\ 0 & 0 & 0 & 0 & L & 0 \\ 0 & 0 & 0 & 0 & 0 & M \end{bmatrix}$$

9 material parameters

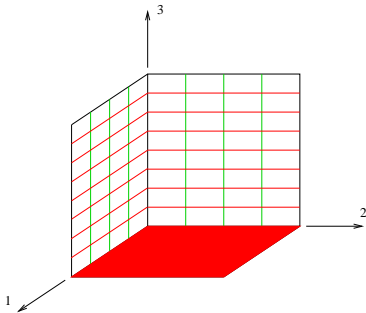
## Quadratic : 2 isotropic directions (here 1 and 2)



$$\underline{\underline{C}} = \begin{bmatrix} A & Q & R & 0 & 0 & 0 \\ Q & A & R & 0 & 0 & 0 \\ R & R & C & 0 & 0 & 0 \\ 0 & 0 & 0 & K & 0 & 0 \\ 0 & 0 & 0 & 0 & L & 0 \\ 0 & 0 & 0 & 0 & 0 & L \end{bmatrix}$$

6 material parameters

## Transversal isotropic : 1 isotropic plane (here 12)



$$\begin{bmatrix} A & Q & R \\ Q & A & R \\ R & R & C \\ & & K \\ & & L \\ & & L \end{bmatrix}$$

## Transversal isotropic : shear test in 12-plane

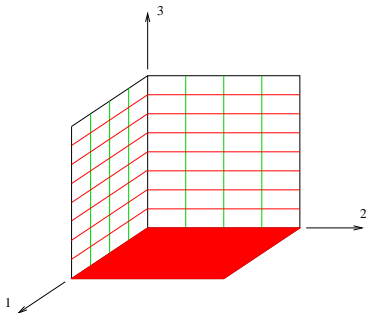
$$\underline{\sigma} = \begin{bmatrix} \sigma_{11} & \sigma_{12} \\ \sigma_{21} & \sigma_{22} \end{bmatrix} = \begin{bmatrix} 0 & \tau \\ \tau & 0 \end{bmatrix} \rightarrow \det(\underline{\sigma} - \sigma \underline{I}) = 0 \rightarrow \begin{cases} \sigma_1 = \tau \\ \sigma_2 = -\tau \end{cases}$$

$$\underline{\varepsilon} = \begin{bmatrix} \varepsilon_{11} & \varepsilon_{12} \\ \varepsilon_{21} & \varepsilon_{22} \end{bmatrix} = \begin{bmatrix} 0 & \frac{1}{2}\gamma \\ \frac{1}{2}\gamma & 0 \end{bmatrix} \rightarrow \det(\underline{\varepsilon} - \varepsilon \underline{I}) = 0 \rightarrow \begin{cases} \varepsilon_1 = \frac{1}{2}\gamma \\ \varepsilon_2 = -\frac{1}{2}\gamma \end{cases}$$

$$\begin{bmatrix} \sigma_1 \\ \sigma_2 \end{bmatrix} = \begin{bmatrix} A & Q \\ Q & A \end{bmatrix} \begin{bmatrix} \varepsilon_1 \\ \varepsilon_2 \end{bmatrix} \rightarrow \begin{aligned} \sigma_1 &= A\varepsilon_1 + Q\varepsilon_2 = \tau = K\gamma \\ \sigma_2 &= Q\varepsilon_1 + A\varepsilon_2 = -\tau = -K\gamma \end{aligned} \rightarrow$$

$$\left. \begin{aligned} (A - Q)(\varepsilon_1 - \varepsilon_2) &= 2K\gamma \\ \varepsilon_1 &= \frac{1}{2}\gamma \quad ; \quad \varepsilon_2 = -\frac{1}{2}\gamma \end{aligned} \right\} \rightarrow \boxed{K = \frac{1}{2}(A - Q)}$$

# Transversal isotropic

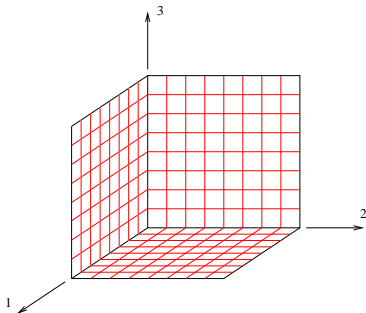


$$\underline{\underline{C}} = \begin{bmatrix} A & Q & R & 0 & 0 & 0 \\ Q & A & R & 0 & 0 & 0 \\ R & R & C & 0 & 0 & 0 \\ 0 & 0 & 0 & K & 0 & 0 \\ 0 & 0 & 0 & 0 & L & 0 \\ 0 & 0 & 0 & 0 & 0 & L \end{bmatrix}$$

$$K = \frac{1}{2}(A - Q)$$

**5** material parameters

## Cubic : 3 isotropic directions (here 1, 2 and 3)

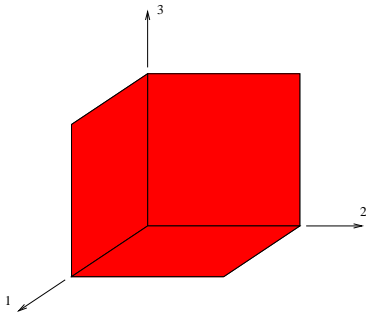


$$\underline{\underline{C}} = \begin{bmatrix} A & Q & Q & 0 & 0 & 0 \\ Q & A & Q & 0 & 0 & 0 \\ Q & Q & A & 0 & 0 & 0 \\ 0 & 0 & 0 & L & 0 & 0 \\ 0 & 0 & 0 & 0 & L & 0 \\ 0 & 0 & 0 & 0 & 0 & L \end{bmatrix}$$

$$L \neq \frac{1}{2}(A - Q)$$

**3** material parameters

# Isotropic



$$\underline{\underline{C}} = \begin{bmatrix} A & Q & Q & 0 & 0 & 0 \\ Q & A & Q & 0 & 0 & 0 \\ Q & Q & A & 0 & 0 & 0 \\ 0 & 0 & 0 & L & 0 & 0 \\ 0 & 0 & 0 & 0 & L & 0 \\ 0 & 0 & 0 & 0 & 0 & L \end{bmatrix}$$

$$L = \frac{1}{2}(A - Q)$$

2 material parameters

# LINEAR ELASTIC ISOTROPIC MATERIAL

## ENGINEERING PARAMETERS

[back to index](#)



# Tensile test

$$\begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{33} \\ \sigma_{12} \\ \sigma_{23} \\ \sigma_{31} \end{bmatrix} = \begin{bmatrix} A & Q & Q & 0 & 0 & 0 \\ Q & A & Q & 0 & 0 & 0 \\ Q & Q & A & 0 & 0 & 0 \\ 0 & 0 & 0 & L & 0 & 0 \\ 0 & 0 & 0 & 0 & L & 0 \\ 0 & 0 & 0 & 0 & 0 & L \end{bmatrix} \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \varepsilon_{33} \\ \gamma_{12} \\ \gamma_{23} \\ \gamma_{31} \end{bmatrix} \quad \text{with} \quad L = \frac{1}{2}(A - Q)$$

$$\underline{\underline{\varepsilon}}^T = [ \varepsilon \quad \varepsilon_d \quad \varepsilon_d \quad 0 \quad 0 \quad 0 ] ; \quad \underline{\underline{\sigma}}^T = [ \sigma \quad 0 \quad 0 \quad 0 \quad 0 \quad 0 ]$$

$$\left. \begin{array}{l} \sigma = A\varepsilon + 2Q\varepsilon_d \\ 0 = Q\varepsilon + (A + Q)\varepsilon_d \rightarrow \varepsilon_d = -\frac{Q}{A+Q}\varepsilon \\ \varepsilon_d = -\nu\varepsilon ; \quad \sigma = E\varepsilon \end{array} \right\} \rightarrow \sigma = \frac{A^2 + AQ - 2Q^2}{A + Q} \varepsilon \quad \left. \vphantom{\begin{array}{l} \sigma = A\varepsilon + 2Q\varepsilon_d \\ 0 = Q\varepsilon + (A + Q)\varepsilon_d \end{array}} \right\} \rightarrow$$

$$A = \frac{(1-\nu)E}{(1+\nu)(1-2\nu)} \quad Q = \frac{\nu E}{(1+\nu)(1-2\nu)} \quad L = \frac{E}{2(1+\nu)}$$

# Shear test

$$\begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{33} \\ \sigma_{12} \\ \sigma_{23} \\ \sigma_{31} \end{bmatrix} = \begin{bmatrix} A & Q & Q & 0 & 0 & 0 \\ Q & A & Q & 0 & 0 & 0 \\ Q & Q & A & 0 & 0 & 0 \\ 0 & 0 & 0 & L & 0 & 0 \\ 0 & 0 & 0 & 0 & L & 0 \\ 0 & 0 & 0 & 0 & 0 & L \end{bmatrix} \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \varepsilon_{33} \\ \gamma_{12} \\ \gamma_{23} \\ \gamma_{31} \end{bmatrix} \quad \text{with} \quad L = \frac{1}{2}(A - Q)$$

$$\underline{\underline{\varepsilon}}^T = [0 \ 0 \ 0 \ 0 \ 0 \ \gamma] ; \underline{\underline{\sigma}}^T = [0 \ 0 \ 0 \ 0 \ 0 \ \tau]$$

$$\tau = L\gamma = \frac{E}{2(1+\nu)}\gamma = G\gamma$$

## Volume change

$$\begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{33} \\ \sigma_{12} \\ \sigma_{23} \\ \sigma_{31} \end{bmatrix} = \begin{bmatrix} A & Q & Q & 0 & 0 & 0 \\ Q & A & Q & 0 & 0 & 0 \\ Q & Q & A & 0 & 0 & 0 \\ 0 & 0 & 0 & L & 0 & 0 \\ 0 & 0 & 0 & 0 & L & 0 \\ 0 & 0 & 0 & 0 & 0 & L \end{bmatrix} \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \varepsilon_{33} \\ \gamma_{12} \\ \gamma_{23} \\ \gamma_{31} \end{bmatrix} \quad \text{with} \quad L = \frac{1}{2}(A - Q)$$

$$\underline{\underline{\varepsilon}}^T = \begin{bmatrix} \varepsilon_{11} & \varepsilon_{22} & \varepsilon_{33} & 0 & 0 & 0 \end{bmatrix}$$

$$\begin{aligned} J - 1 \approx \varepsilon_{11} + \varepsilon_{22} + \varepsilon_{33} &= \frac{1 - 2\nu}{E} (\sigma_{11} + \sigma_{22} + \sigma_{33}) \\ &= \frac{3(1 - 2\nu)}{E} \frac{1}{3} (\sigma_{11} + \sigma_{22} + \sigma_{33}) = \frac{1}{K} \frac{1}{3} \text{tr}(\underline{\underline{\sigma}}) \end{aligned}$$

# Isotropic compliance and stiffness matrix

$$\begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \varepsilon_{33} \\ \gamma_{12} \\ \gamma_{23} \\ \gamma_{31} \end{bmatrix} = \frac{1}{E} \begin{bmatrix} 1 & -\nu & -\nu & 0 & 0 & 0 \\ -\nu & 1 & -\nu & 0 & 0 & 0 \\ -\nu & -\nu & 1 & 0 & 0 & 0 \\ 0 & 0 & 0 & 2(1+\nu) & 0 & 0 \\ 0 & 0 & 0 & 0 & 2(1+\nu) & 0 \\ 0 & 0 & 0 & 0 & 0 & 2(1+\nu) \end{bmatrix} \begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{33} \\ \sigma_{12} \\ \sigma_{23} \\ \sigma_{31} \end{bmatrix}$$

$$\begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{33} \\ \sigma_{12} \\ \sigma_{23} \\ \sigma_{31} \end{bmatrix} = \alpha \begin{bmatrix} 1-\nu & \nu & \nu & 0 & 0 & 0 \\ \nu & 1-\nu & \nu & 0 & 0 & 0 \\ \nu & \nu & 1-\nu & 0 & 0 & 0 \\ 0 & 0 & 0 & \frac{1}{2}(1-2\nu) & 0 & 0 \\ 0 & 0 & 0 & 0 & \frac{1}{2}(1-2\nu) & 0 \\ 0 & 0 & 0 & 0 & 0 & \frac{1}{2}(1-2\nu) \end{bmatrix} \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \varepsilon_{33} \\ \gamma_{12} \\ \gamma_{23} \\ \gamma_{31} \end{bmatrix}$$

$$\text{with } \alpha = \frac{E}{(1+\nu)(1-2\nu)}$$

# LINEAR ELASTIC ISOTROPIC MATERIAL

## TENSORIAL FORM

[back to index](#)

## Column/matrix notation of Hooke's law

$$\begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \varepsilon_{33} \\ \varepsilon_{12} \\ \varepsilon_{23} \\ \varepsilon_{31} \end{bmatrix} = \frac{1}{E} \begin{bmatrix} 1 & -\nu & -\nu & 0 & 0 & 0 \\ -\nu & 1 & -\nu & 0 & 0 & 0 \\ -\nu & -\nu & 1 & 0 & 0 & 0 \\ 0 & 0 & 0 & 1+\nu & 0 & 0 \\ 0 & 0 & 0 & 0 & 1+\nu & 0 \\ 0 & 0 & 0 & 0 & 0 & 1+\nu \end{bmatrix} \begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{33} \\ \sigma_{12} \\ \sigma_{23} \\ \sigma_{31} \end{bmatrix}$$

$$\begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{33} \\ \sigma_{12} \\ \sigma_{23} \\ \sigma_{31} \end{bmatrix} = \alpha \begin{bmatrix} 1-\nu & \nu & \nu & 0 & 0 & 0 \\ \nu & 1-\nu & \nu & 0 & 0 & 0 \\ \nu & \nu & 1-\nu & 0 & 0 & 0 \\ 0 & 0 & 0 & 1-2\nu & 0 & 0 \\ 0 & 0 & 0 & 0 & 1-2\nu & 0 \\ 0 & 0 & 0 & 0 & 0 & 1-2\nu \end{bmatrix} \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \varepsilon_{33} \\ \varepsilon_{12} \\ \varepsilon_{23} \\ \varepsilon_{31} \end{bmatrix}$$

$$\text{with } \alpha = \frac{E}{(1+\nu)(1-2\nu)}$$

# Isotropic stiffness matrix

$$\begin{aligned}
 \begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{33} \\ \sigma_{12} \\ \sigma_{23} \\ \sigma_{31} \end{bmatrix} &= \frac{E}{(1+\nu)} \begin{bmatrix} \frac{(1-\nu)}{(1-2\nu)} & \frac{\nu}{(1-2\nu)} & \frac{\nu}{(1-2\nu)} & 0 & 0 & 0 \\ \frac{\nu}{(1-2\nu)} & \frac{(1-\nu)}{(1-2\nu)} & \frac{\nu}{(1-2\nu)} & 0 & 0 & 0 \\ \frac{\nu}{(1-2\nu)} & \frac{\nu}{(1-2\nu)} & \frac{(1-\nu)}{(1-2\nu)} & 0 & 0 & 0 \\ \frac{(1-\nu)}{(1-2\nu)} & \frac{\nu}{(1-2\nu)} & \frac{\nu}{(1-2\nu)} & 1 & 0 & 0 \\ 0 & 0 & 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 0 & 0 & 1 \end{bmatrix} \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \varepsilon_{33} \\ \varepsilon_{12} \\ \varepsilon_{23} \\ \varepsilon_{31} \end{bmatrix} \\
 &= \frac{E}{(1+\nu)} \left[ \begin{bmatrix} \frac{(1-\nu)}{(1-2\nu)} - 1 & \frac{\nu}{(1-2\nu)} & \frac{\nu}{(1-2\nu)} & 0 & 0 & 0 \\ \frac{\nu}{(1-2\nu)} & \frac{(1-\nu)}{(1-2\nu)} - 1 & \frac{\nu}{(1-2\nu)} & 0 & 0 & 0 \\ \frac{\nu}{(1-2\nu)} & \frac{\nu}{(1-2\nu)} & \frac{(1-\nu)}{(1-2\nu)} - 1 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 \end{bmatrix} + \begin{bmatrix} 1 & 0 & 0 & 0 & 0 & 0 \\ 0 & 1 & 0 & 0 & 0 & 0 \\ 0 & 0 & 1 & 0 & 0 & 0 \\ 0 & 0 & 0 & 1 & 0 & 0 \\ 0 & 0 & 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 0 & 0 & 1 \end{bmatrix} \right] \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \varepsilon_{33} \\ \varepsilon_{12} \\ \varepsilon_{23} \\ \varepsilon_{31} \end{bmatrix} \\
 &= \left[ \frac{E\nu}{(1+\nu)(1-2\nu)} \begin{bmatrix} 1 & 1 & 1 & 0 & 0 & 0 \\ 1 & 1 & 1 & 0 & 0 & 0 \\ 1 & 1 & 1 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 \end{bmatrix} + \right. \\
 &\quad \left. \frac{E}{(1+\nu)} \begin{bmatrix} 1 & 0 & 0 & 0 & 0 & 0 \\ 0 & 1 & 0 & 0 & 0 & 0 \\ 0 & 0 & 1 & 0 & 0 & 0 \\ 0 & 0 & 0 & 1 & 0 & 0 \\ 0 & 0 & 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 0 & 0 & 1 \end{bmatrix} \right] \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \varepsilon_{33} \\ \varepsilon_{12} \\ \varepsilon_{23} \\ \varepsilon_{31} \end{bmatrix}
 \end{aligned}$$

# Isotropic stiffness tensor

$$\boldsymbol{\sigma} = \left[ \frac{E\nu}{(1+\nu)(1-2\nu)} \right] \mathbf{I} \operatorname{tr}(\boldsymbol{\varepsilon}) + \left[ \frac{E}{(1+\nu)} \right] \boldsymbol{\varepsilon}$$

$$= Q \operatorname{tr}(\boldsymbol{\varepsilon}) \mathbf{I} + 2L \boldsymbol{\varepsilon}$$

$$= c_0 \operatorname{tr}(\boldsymbol{\varepsilon}) \mathbf{I} + c_1 \boldsymbol{\varepsilon}$$

$$= \left[ c_0 \mathbf{I} \mathbf{I} + c_1 {}^4\mathbf{I}^s \right] : \boldsymbol{\varepsilon}$$

$$\text{with } {}^4\mathbf{I}^s = \frac{1}{2} ({}^4\mathbf{I} + {}^4\mathbf{I}^{rc})$$

$$= {}^4\mathbf{C} : \boldsymbol{\varepsilon}$$



# Stiffness and compliance tensor

$$\boldsymbol{\sigma} = {}^4\mathbf{C} : \boldsymbol{\varepsilon}$$

$$= \left[ c_0 \mathbb{I} + c_1 {}^4\mathbf{I}^s \right] : \boldsymbol{\varepsilon}$$

$$\text{with } {}^4\mathbf{I}^s = \frac{1}{2} ({}^4\mathbf{I} + {}^4\mathbf{I}^{rc})$$

$$= c_0 \text{tr}(\boldsymbol{\varepsilon}) \mathbb{I} + c_1 \boldsymbol{\varepsilon}$$

$$= c_0 \text{tr}(\boldsymbol{\varepsilon}) \mathbb{I} + c_1 \left\{ \boldsymbol{\varepsilon}^d + \frac{1}{3} \text{tr}(\boldsymbol{\varepsilon}) \mathbb{I} \right\}$$

$$= (c_0 + \frac{1}{3} c_1) \text{tr}(\boldsymbol{\varepsilon}) \mathbb{I} + c_1 \boldsymbol{\varepsilon}^d$$

$$= (3c_0 + c_1) \frac{1}{3} \text{tr}(\boldsymbol{\varepsilon}) \mathbb{I} + c_1 \boldsymbol{\varepsilon}^d$$

$$= (3c_0 + c_1) \boldsymbol{\varepsilon}^h + c_1 \boldsymbol{\varepsilon}^d$$

$$= \boldsymbol{\sigma}^h + \boldsymbol{\sigma}^d$$

$$c_0 = \frac{\nu E}{(1 + \nu)(1 - 2\nu)} = Q$$

$$\gamma_0 = -\frac{c_0}{(3c_0 + c_1)c_1} = -\frac{\nu}{E} = q$$

$$\boldsymbol{\varepsilon} = \boldsymbol{\varepsilon}^h + \boldsymbol{\varepsilon}^d$$

$$= \frac{1}{3c_0 + c_1} \boldsymbol{\sigma}^h + \frac{1}{c_1} \boldsymbol{\sigma}^d$$

$$= \frac{1}{3c_0 + c_1} \frac{1}{3} \text{tr}(\boldsymbol{\sigma}) \mathbb{I} + \frac{1}{c_1} \left\{ \boldsymbol{\sigma} - \frac{1}{3} \text{tr}(\boldsymbol{\sigma}) \mathbb{I} \right\}$$

$$= -\frac{c_0}{(3c_0 + c_1)c_1} \text{tr}(\boldsymbol{\sigma}) \mathbb{I} + \frac{1}{c_1} \boldsymbol{\sigma}$$

$$= \left[ -\frac{c_0}{(3c_0 + c_1)c_1} \mathbb{I} + \frac{1}{c_1} {}^4\mathbf{I}^s \right] : \boldsymbol{\sigma}$$

$$= \left[ \gamma_0 \mathbb{I} + \gamma_1 {}^4\mathbf{I}^s \right] : \boldsymbol{\sigma}$$

$$= {}^4\mathbf{S} : \boldsymbol{\sigma}$$

$$c_1 = \frac{E}{1 + \nu} = 2L$$

$$\gamma_1 = \frac{1}{c_1} = \frac{1 + \nu}{E} = \frac{1}{2} I$$

# Stiffness and compliance components

$$\boldsymbol{\sigma} = [c_0 \mathbf{1}\mathbf{1} + c_1 {}^4\mathbf{I}^s] : \boldsymbol{\varepsilon}$$

$$\sigma_{ij} = [c_0 \delta_{ij} \delta_{kl} + c_1 \frac{1}{2} (\delta_{il} \delta_{jk} + \delta_{ik} \delta_{jl})] \varepsilon_{lk}$$

$$= c_0 \delta_{ij} \varepsilon_{kk} + c_1 \varepsilon_{ij}$$

$$= c_1 \left( \varepsilon_{ij} + \frac{c_0}{c_1} \delta_{ij} \varepsilon_{kk} \right)$$

$$= \frac{E}{1+\nu} \left( \varepsilon_{ij} + \frac{\nu}{1-2\nu} \delta_{ij} \varepsilon_{kk} \right)$$

$$\boldsymbol{\varepsilon} = \left[ -\frac{c_0}{(3c_0 + c_1)c_1} \mathbf{1}\mathbf{1} + \frac{1}{c_1} {}^4\mathbf{I}^s \right] : \boldsymbol{\sigma}$$

$$\varepsilon_{ij} = \left[ -\frac{c_0}{(3c_0 + c_1)c_1} \delta_{ij} \delta_{kl} + \frac{1}{c_1} \frac{1}{2} (\delta_{il} \delta_{jk} + \delta_{ik} \delta_{jl}) \right] \sigma_{lk}$$

$$= -\frac{c_0}{(3c_0 + c_1)c_1} \delta_{ij} \sigma_{kk} + \frac{1}{c_1} \sigma_{ij}$$

$$= \frac{1}{c_1} \left( \sigma_{ij} - \frac{c_0}{3c_0 + c_1} \delta_{ij} \sigma_{kk} \right)$$

$$= \frac{1+\nu}{E} \left( \sigma_{ij} - \frac{\nu}{1+\nu} \delta_{ij} \sigma_{kk} \right)$$

# Specific elastic energy

$$W = \frac{1}{2} \boldsymbol{\sigma} : \boldsymbol{\varepsilon} = \frac{1}{2} \boldsymbol{\sigma} : {}^4\mathbf{S} : \boldsymbol{\sigma} = \frac{1}{2} (\boldsymbol{\sigma}^h + \boldsymbol{\sigma}^d) : {}^4\mathbf{S} : (\boldsymbol{\sigma}^h + \boldsymbol{\sigma}^d)$$

$$= \frac{1}{2} (\boldsymbol{\sigma}^h + \boldsymbol{\sigma}^d) : \left( \gamma_0 \mathbf{I} + \gamma_1 {}^4\mathbf{I}^s \right) : (\boldsymbol{\sigma}^h + \boldsymbol{\sigma}^d)$$

$$\gamma_0 \mathbf{I} [\mathbf{I} : \boldsymbol{\sigma}^h] = \gamma_0 \mathbf{I} [\mathbf{I} : \frac{1}{3} \text{tr}(\boldsymbol{\sigma})] = \gamma_0 \mathbf{I} [\text{tr}(\boldsymbol{\sigma})] = 3\gamma_0 \boldsymbol{\sigma}^h$$

$$\gamma_0 \mathbf{I} [\mathbf{I} : \boldsymbol{\sigma}^d] = \gamma_0 \mathbf{I} [\text{tr}(\boldsymbol{\sigma}^d)] = \gamma_0 \mathbf{I} [0] = 0$$

$$= \frac{1}{2} (\boldsymbol{\sigma}^h + \boldsymbol{\sigma}^d) : (3\gamma_0 \boldsymbol{\sigma}^h + \gamma_1 \boldsymbol{\sigma}^h + \gamma_1 \boldsymbol{\sigma}^d)$$

$$\boldsymbol{\sigma}^h : \boldsymbol{\sigma}^h = \frac{1}{3} \text{tr}(\boldsymbol{\sigma}) \mathbf{I} : \frac{1}{3} \text{tr}(\boldsymbol{\sigma}) \mathbf{I} = \frac{1}{9} \text{tr}^2(\boldsymbol{\sigma}) (3) = \frac{1}{3} \text{tr}^2(\boldsymbol{\sigma})$$

$$\boldsymbol{\sigma}^h : \boldsymbol{\sigma}^d = \frac{1}{3} \text{tr}(\boldsymbol{\sigma}) \mathbf{I} : [\boldsymbol{\sigma} - \frac{1}{3} \text{tr}(\boldsymbol{\sigma}) \mathbf{I}] = \frac{1}{3} \text{tr}^2(\boldsymbol{\sigma}) - \frac{1}{3} \text{tr}^2(\boldsymbol{\sigma}) = 0$$

$$= \left[ \frac{1}{2} (\gamma_0 + \frac{1}{3} \gamma_1) \right] \text{tr}^2(\boldsymbol{\sigma}) + \left[ \frac{1}{2} \gamma_1 \right] \boldsymbol{\sigma}^d : \boldsymbol{\sigma}^d$$

$$= \left[ \frac{1}{2} \frac{1-2\nu}{3E} \right] \text{tr}^2(\boldsymbol{\sigma}) + \left[ \frac{1}{2} \frac{1+\nu}{E} \right] \boldsymbol{\sigma}^d : \boldsymbol{\sigma}^d = \frac{1}{18K} \text{tr}^2(\boldsymbol{\sigma}) + \frac{1}{4G} \boldsymbol{\sigma}^d : \boldsymbol{\sigma}^d$$

$$= W^h + W^d$$

# THERMO-ELASTICITY

[back to index](#)

# Thermoelasticity

## Anisotropic

$$\boldsymbol{\varepsilon} = \boldsymbol{\varepsilon}_m + \boldsymbol{\varepsilon}_T = {}^4\mathbf{S} : \boldsymbol{\sigma} + \mathbf{A}\Delta T \quad \rightarrow$$

$$\underline{\underline{\boldsymbol{\varepsilon}}} = \underline{\underline{\boldsymbol{\varepsilon}}}_m + \underline{\underline{\boldsymbol{\varepsilon}}}_T = \underline{\underline{\mathbf{S}}}\underline{\underline{\boldsymbol{\sigma}}} + \underline{\underline{\mathbf{A}}}\Delta T$$

$$\boldsymbol{\sigma} = {}^4\mathbf{C} : (\boldsymbol{\varepsilon} - \mathbf{A}\Delta T) \quad \rightarrow$$

$$\underline{\underline{\boldsymbol{\sigma}}} = \underline{\underline{\mathbf{C}}}(\underline{\underline{\boldsymbol{\varepsilon}}} - \underline{\underline{\mathbf{A}}}\Delta T)$$

## Isotropic

$$\boldsymbol{\varepsilon} = {}^4\mathbf{S} : \boldsymbol{\sigma} + \alpha \Delta T \mathbf{I} \quad \rightarrow \quad \underline{\underline{\boldsymbol{\varepsilon}}} = \underline{\underline{\mathbf{S}}}\underline{\underline{\boldsymbol{\sigma}}} + \alpha \Delta T \underline{\underline{\mathbf{I}}}$$

$$\boldsymbol{\sigma} = {}^4\mathbf{C} : (\boldsymbol{\varepsilon} - \alpha \Delta T \mathbf{I}) \quad \rightarrow \quad \underline{\underline{\boldsymbol{\sigma}}} = \underline{\underline{\mathbf{C}}}(\underline{\underline{\boldsymbol{\varepsilon}}} - \alpha \Delta T \underline{\underline{\mathbf{I}}})$$

# Orthotropic thermo-elasticity

$$\begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \varepsilon_{33} \\ \gamma_{12} \\ \gamma_{23} \\ \gamma_{31} \end{bmatrix} = \begin{bmatrix} a & q & r & 0 & 0 & 0 \\ q & b & s & 0 & 0 & 0 \\ r & s & c & 0 & 0 & 0 \\ 0 & 0 & 0 & k & 0 & 0 \\ 0 & 0 & 0 & 0 & l & 0 \\ 0 & 0 & 0 & 0 & 0 & m \end{bmatrix} \begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{33} \\ \sigma_{12} \\ \sigma_{23} \\ \sigma_{31} \end{bmatrix} + \alpha \Delta T \begin{bmatrix} 1 \\ 1 \\ 1 \\ 0 \\ 0 \\ 0 \end{bmatrix}$$

$$\begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{33} \\ \sigma_{12} \\ \sigma_{23} \\ \sigma_{31} \end{bmatrix} = \begin{bmatrix} A & Q & R & 0 & 0 & 0 \\ Q & B & S & 0 & 0 & 0 \\ R & S & C & 0 & 0 & 0 \\ 0 & 0 & 0 & K & 0 & 0 \\ 0 & 0 & 0 & 0 & L & 0 \\ 0 & 0 & 0 & 0 & 0 & M \end{bmatrix} \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \varepsilon_{33} \\ \gamma_{12} \\ \gamma_{23} \\ \gamma_{31} \end{bmatrix} - \alpha \Delta T \begin{bmatrix} A + Q + R \\ Q + B + S \\ R + S + C \\ 0 \\ 0 \\ 0 \end{bmatrix}$$

# PLANAR DEFORMATION

[back to index](#)

# Plane strain

$$\varepsilon_{33} = \gamma_{23} = \gamma_{31} = 0 \quad \rightarrow \quad \sigma_{33} = R\varepsilon_{11} + S\varepsilon_{22}$$

$$\underline{\underline{\sigma}} = \begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{12} \end{bmatrix} = \begin{bmatrix} A & Q & 0 \\ Q & B & 0 \\ 0 & 0 & K \end{bmatrix} \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \gamma_{12} \end{bmatrix} = \begin{bmatrix} A_{\varepsilon} & Q_{\varepsilon} & 0 \\ Q_{\varepsilon} & B_{\varepsilon} & 0 \\ 0 & 0 & K \end{bmatrix} \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \gamma_{12} \end{bmatrix} = \underline{\underline{C}}_{\varepsilon} \underline{\underline{\varepsilon}}$$

$$\begin{aligned} \underline{\underline{\varepsilon}} = \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \gamma_{12} \end{bmatrix} &= \frac{1}{AB - Q^2} \begin{bmatrix} B & -Q & 0 \\ -Q & A & 0 \\ 0 & 0 & \frac{AB - Q^2}{K} \end{bmatrix} \begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{12} \end{bmatrix} \\ &= \begin{bmatrix} a_{\varepsilon} & q_{\varepsilon} & 0 \\ q_{\varepsilon} & b_{\varepsilon} & 0 \\ 0 & 0 & k \end{bmatrix} \begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{12} \end{bmatrix} = \underline{\underline{S}}_{\varepsilon} \underline{\underline{\sigma}} \end{aligned}$$

$$\sigma_{33} = \frac{1}{AB^2 - Q^2} [(BR - QS)\sigma_{11} + (AS - QR)\sigma_{22}]$$



# Plane strain

$$\varepsilon_{33} = 0 = r\sigma_{11} + s\sigma_{22} + c\sigma_{33} \quad \rightarrow \quad \sigma_{33} = -\frac{r}{c}\sigma_{11} - \frac{s}{c}\sigma_{22}$$

$$\underline{\underline{\varepsilon}} = \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \gamma_{12} \end{bmatrix} = \begin{bmatrix} a & q & 0 \\ q & b & 0 \\ 0 & 0 & k \end{bmatrix} \begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{12} \end{bmatrix} - \begin{bmatrix} r \\ s \\ 0 \end{bmatrix} \begin{bmatrix} \frac{r}{c} & \frac{s}{c} & 0 \end{bmatrix} \begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{12} \end{bmatrix}$$

$$= \frac{1}{c} \begin{bmatrix} ac - r^2 & qc - rs & 0 \\ qc - sr & bc - s^2 & 0 \\ 0 & 0 & kc \end{bmatrix} \begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{12} \end{bmatrix} = \begin{bmatrix} a_\varepsilon & q_\varepsilon & 0 \\ q_\varepsilon & b_\varepsilon & 0 \\ 0 & 0 & k \end{bmatrix} \begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{12} \end{bmatrix} = \underline{\underline{S}}_\varepsilon \underline{\underline{\sigma}}$$

$$\underline{\underline{\sigma}} = \begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{12} \end{bmatrix} = \begin{bmatrix} a_\varepsilon & q_\varepsilon & 0 \\ q_\varepsilon & b_\varepsilon & 0 \\ 0 & 0 & k \end{bmatrix}^{-1} \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \gamma_{12} \end{bmatrix} = \frac{1}{\Delta_s} \begin{bmatrix} bc - s^2 & -qc + rs & 0 \\ -qc + rs & ac - r^2 & 0 \\ 0 & 0 & \frac{\Delta_s}{k} \end{bmatrix} \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \varepsilon_{12} \end{bmatrix}$$

$$\text{with } \Delta_s = abc - as^2 - br^2 - cq^2 + 2qrs$$

$$= \begin{bmatrix} A_\varepsilon & Q_\varepsilon & 0 \\ Q_\varepsilon & B_\varepsilon & 0 \\ 0 & 0 & K \end{bmatrix} \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \varepsilon_{12} \end{bmatrix} = \underline{\underline{C}}_\varepsilon \underline{\underline{\sigma}}$$

$$\sigma_{33} = -\frac{1}{\Delta_s} [(br - qs)\varepsilon_{11} + (as - qr)\varepsilon_{22}]$$

# Plane stress

$$\sigma_{33} = \sigma_{23} = \sigma_{31} = 0 \quad \rightarrow \quad \varepsilon_{33} = r\sigma_{11} + s\sigma_{22}$$

$$\underline{\underline{\varepsilon}} = \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \gamma_{12} \end{bmatrix} = \begin{bmatrix} a & q & 0 \\ q & b & 0 \\ 0 & 0 & k \end{bmatrix} \begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{12} \end{bmatrix} = \begin{bmatrix} a_{\sigma} & q_{\sigma} & 0 \\ q_{\sigma} & b_{\sigma} & 0 \\ 0 & 0 & k \end{bmatrix} \begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{12} \end{bmatrix} = \underline{\underline{S}}_{\sigma} \underline{\underline{\sigma}}$$

$$\underline{\underline{\sigma}} = \begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{12} \end{bmatrix} = \frac{1}{ab - q^2} \begin{bmatrix} b & -q & 0 \\ -q & a & 0 \\ 0 & 0 & \frac{ab - q^2}{k} \end{bmatrix} \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \gamma_{12} \end{bmatrix}$$

$$= \begin{bmatrix} A_{\sigma} & Q_{\sigma} & 0 \\ Q_{\sigma} & B_{\sigma} & 0 \\ 0 & 0 & K \end{bmatrix} \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \gamma_{12} \end{bmatrix} = \underline{\underline{C}}_{\sigma} \underline{\underline{\varepsilon}}$$

$$\varepsilon_{33} = \frac{1}{ab - q^2} [(br - qs)\varepsilon_{11} + (as - qr)\varepsilon_{22}]$$

# Plane stress

$$\sigma_{33} = 0 = R\varepsilon_{11} + S\varepsilon_{22} + C\varepsilon_{33} \rightarrow \varepsilon_{33} = -\frac{R}{C}\varepsilon_{11} - \frac{S}{C}\varepsilon_{22}$$

$$\begin{aligned} \underline{\sigma} = \begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{12} \end{bmatrix} &= \begin{bmatrix} A & Q & 0 \\ Q & B & 0 \\ 0 & 0 & K \end{bmatrix} \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \gamma_{12} \end{bmatrix} - \begin{bmatrix} R \\ S \\ 0 \end{bmatrix} \left[ \frac{R}{C} \quad \frac{S}{C} \quad 0 \right] \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \gamma_{12} \end{bmatrix} \\ &= \frac{1}{C} \begin{bmatrix} AC - R^2 & QC - RS & 0 \\ QC - RS & BC - S^2 & 0 \\ 0 & 0 & KC \end{bmatrix} \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \gamma_{12} \end{bmatrix} \\ &= \begin{bmatrix} A_{\sigma} & Q_{\sigma} & 0 \\ Q_{\sigma} & B_{\sigma} & 0 \\ 0 & 0 & K \end{bmatrix} \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \gamma_{12} \end{bmatrix} = \underline{\underline{C}}_{\sigma} \underline{\varepsilon} \end{aligned}$$

$$\underline{\varepsilon} = \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \gamma_{12} \end{bmatrix} = \begin{bmatrix} A_{\sigma} & Q_{\sigma} & 0 \\ Q_{\sigma} & B_{\sigma} & 0 \\ 0 & 0 & K \end{bmatrix}^{-1} \begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{12} \end{bmatrix} = \frac{1}{\Delta_c} \begin{bmatrix} BC - S^2 & -QC + RS & 0 \\ -QC + RS & AC - R^2 & 0 \\ 0 & 0 & \frac{\Delta_c}{K} \end{bmatrix} \begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{12} \end{bmatrix}$$

$$\text{with } \Delta_c = ABC - AS^2 - BR^2 - CQ^2 + 2QRS$$

$$= \begin{bmatrix} a_{\sigma} & q_{\sigma} & 0 \\ q_{\sigma} & b_{\sigma} & 0 \\ 0 & 0 & k \end{bmatrix} = \underline{\underline{S}}_{\sigma} \underline{\sigma}$$

## Plane strain thermo-elastic

$$\sigma_{33} = R\varepsilon_{11} + S\varepsilon_{22} - \alpha(R + S + C)\Delta T \quad (\text{from } \underline{\underline{C}})$$

$$= -\frac{r}{c}\sigma_{11} - \frac{s}{c}\sigma_{22} - \frac{\alpha}{c}\Delta T \quad (\text{from } \underline{\underline{S}})$$

$$\begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{12} \end{bmatrix} = \begin{bmatrix} A & Q & 0 \\ Q & B & 0 \\ 0 & 0 & K \end{bmatrix} \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \gamma_{12} \end{bmatrix} - \alpha\Delta T \begin{bmatrix} A + Q + R \\ B + Q + S \\ 0 \end{bmatrix}$$

$$\begin{aligned} \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \gamma_{12} \end{bmatrix} &= \begin{bmatrix} A & Q & 0 \\ Q & B & 0 \\ 0 & 0 & K \end{bmatrix}^{-1} \left\{ \begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{12} \end{bmatrix} + \alpha\Delta T \begin{bmatrix} A + Q + R \\ B + Q + S \\ 0 \end{bmatrix} \right\} \\ &= \begin{bmatrix} a_\varepsilon & q_\varepsilon & 0 \\ q_\varepsilon & b_\varepsilon & 0 \\ 0 & 0 & k \end{bmatrix} \begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{12} \end{bmatrix} + \alpha\Delta T \begin{bmatrix} 1 + q_\varepsilon S + a_\varepsilon R \\ 1 + q_\varepsilon R + b_\varepsilon S \\ 0 \end{bmatrix} \end{aligned}$$

# Plane stress thermo-elastic

$$\varepsilon_{33} = r\sigma_{11} + s\sigma_{22} + \alpha\Delta T \quad (\text{from } \underline{S})$$

$$= -\frac{R}{C}\varepsilon_{11} - \frac{S}{C}\varepsilon_{22} + \frac{1}{C}(R + S + C)\alpha\Delta T \quad (\text{from } \underline{C})$$

$$\begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \gamma_{12} \end{bmatrix} = \begin{bmatrix} a & q & 0 \\ q & b & 0 \\ 0 & 0 & k \end{bmatrix} \begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{12} \end{bmatrix} + \alpha\Delta T \begin{bmatrix} 1 \\ 1 \\ 0 \end{bmatrix}$$

$$\begin{aligned} \begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{12} \end{bmatrix} &= \begin{bmatrix} a & q & 0 \\ q & b & 0 \\ 0 & 0 & k \end{bmatrix}^{-1} \left\{ \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \gamma_{12} \end{bmatrix} - \alpha\Delta T \begin{bmatrix} 1 \\ 1 \\ 0 \end{bmatrix} \right\} \\ &= \begin{bmatrix} A_{\sigma} & Q_{\sigma} & 0 \\ Q_{\sigma} & B_{\sigma} & 0 \\ 0 & 0 & K \end{bmatrix} \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \gamma_{12} \end{bmatrix} - \alpha\Delta T \begin{bmatrix} A_{\sigma} + Q_{\sigma} \\ B_{\sigma} + Q_{\sigma} \\ 0 \end{bmatrix} \end{aligned}$$

# General planar material laws

$$\underline{\underline{C}}_p = \begin{bmatrix} A_p & Q_p & 0 \\ Q_p & B_p & 0 \\ 0 & 0 & K \end{bmatrix} - \alpha \Delta T \begin{bmatrix} \Theta_{p1} \\ \Theta_{p2} \\ 0 \end{bmatrix}$$

$$\underline{\underline{S}}_p = \begin{bmatrix} a_p & q_p & 0 \\ q_p & b_p & 0 \\ 0 & 0 & k \end{bmatrix} + \alpha \Delta T \begin{bmatrix} \theta_{p1} \\ \theta_{p2} \\ 0 \end{bmatrix}$$

plane strain :  $(\ )_p = (\ )_\varepsilon$

plane stress :  $(\ )_p = (\ )_\sigma$

## ELASTIC LIMIT

[back to index](#)

# Elastic limit criteria

failure mode	mechanism
plastic yielding	crystallographic slip (metals)
brittle fracture	(sudden) breakage of bonds
progressive damage	micro-cracks → growth → coalescence
fatigue	damage/fracture under cyclic loading
dynamic failure	vibration → resonance
thermal failure	creep / melting
elastic instabilities	buckling → plastic deformation

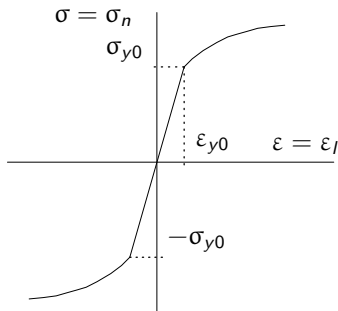


## Yield function : one-dimensional

$$f(\sigma) = \sigma^2 - \sigma_{y0}^2 = 0 \quad \rightarrow$$

$$g(\sigma) = \sigma^2 = \sigma_{y0}^2 = g_t$$

$g_t$  = limit in tensile test



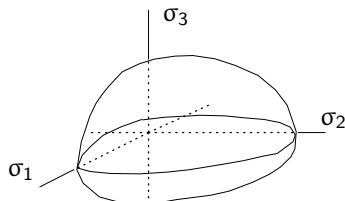
# Yield function : three-dimensional

$$f(\boldsymbol{\sigma}) = 0 \quad \rightarrow \quad g(\boldsymbol{\sigma}) = g_t$$

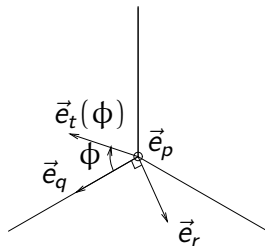
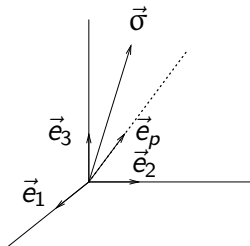
yield surface in 6D stress space

$$f(\sigma_1, \sigma_2, \sigma_3) = 0 \quad \rightarrow \quad g(\sigma_1, \sigma_2, \sigma_3) = g_t$$

yield surface in 3D principal stress space



# Principal stress space



hydrostatic axis

$$\vec{e}_p = \frac{1}{3}\sqrt{3}(\vec{e}_1 + \vec{e}_2 + \vec{e}_3) \quad \text{with} \quad \|\vec{e}_p\| = 1$$

plane  $\perp$  hydrostatic axis

$$\vec{e}_q^* = \vec{e}_1 - (\vec{e}_p \cdot \vec{e}_1)\vec{e}_p = \vec{e}_1 - \frac{1}{3}(\vec{e}_1 + \vec{e}_2 + \vec{e}_3) = \frac{1}{3}(2\vec{e}_1 - \vec{e}_2 - \vec{e}_3)$$

$$\vec{e}_q = \frac{1}{\sqrt{6}}\sqrt{6}(2\vec{e}_1 - \vec{e}_2 - \vec{e}_3)$$

$$\vec{e}_r = \vec{e}_p * \vec{e}_q = \frac{1}{3}\sqrt{3}(\vec{e}_1 + \vec{e}_2 + \vec{e}_3) * \frac{1}{\sqrt{6}}\sqrt{6}(2\vec{e}_1 - \vec{e}_2 - \vec{e}_3) = \frac{1}{2}\sqrt{2}(\vec{e}_2 - \vec{e}_3)$$

vector in  $\Pi$ -plane

$$\vec{e}_t(\phi) = \cos(\phi)\vec{e}_q - \sin(\phi)\vec{e}_r$$

# Principal stress space

$$\vec{\sigma} = \sigma_1 \vec{e}_1 + \sigma_2 \vec{e}_2 + \sigma_3 \vec{e}_3 = \vec{\sigma}^h + \vec{\sigma}^d$$

$$\vec{\sigma}^h = (\vec{\sigma} \cdot \vec{e}_p) \vec{e}_p = \sigma^h \vec{e}_p = \frac{1}{3} \sqrt{3} (\sigma_1 + \sigma_2 + \sigma_3) \vec{e}_p = \sqrt{3} \sigma_m \vec{e}_p$$

$$\sigma^h = \frac{1}{3} \sqrt{3} (\sigma_1 + \sigma_2 + \sigma_3)$$

$$\vec{\sigma}^d = \vec{\sigma} - (\vec{\sigma} \cdot \vec{e}_p) \vec{e}_p$$

$$= \sigma_1 \vec{e}_1 + \sigma_2 \vec{e}_2 + \sigma_3 \vec{e}_3 - \frac{1}{3} \sqrt{3} (\sigma_1 + \sigma_2 + \sigma_3) \frac{1}{\sqrt{3}} (\vec{e}_1 + \vec{e}_2 + \vec{e}_3)$$

$$= \sigma_1 \vec{e}_1 + \sigma_2 \vec{e}_2 + \sigma_3 \vec{e}_3 -$$

$$\frac{1}{3} (\sigma_1 \vec{e}_1 + \sigma_2 \vec{e}_1 + \sigma_3 \vec{e}_1 + \sigma_1 \vec{e}_2 + \sigma_2 \vec{e}_2 + \sigma_3 \vec{e}_2 + \sigma_1 \vec{e}_3 + \sigma_2 \vec{e}_3 + \sigma_3 \vec{e}_3)$$

$$= \frac{1}{3} \{ (2\sigma_1 - \sigma_2 - \sigma_3) \vec{e}_1 + (-\sigma_1 + 2\sigma_2 - \sigma_3) \vec{e}_2 + (-\sigma_1 - \sigma_2 + 2\sigma_3) \vec{e}_3 \}$$

$$\sigma^d = \|\vec{\sigma}^d\| = \sqrt{\vec{\sigma}^d \cdot \vec{\sigma}^d}$$

$$= \frac{1}{3} \sqrt{(2\sigma_1 - \sigma_2 - \sigma_3)^2 + (-\sigma_1 + 2\sigma_2 - \sigma_3)^2 + (-\sigma_1 - \sigma_2 + 2\sigma_3)^2}$$

$$= \sqrt{\frac{2}{3} (\sigma_1^2 + \sigma_2^2 + \sigma_3^2 - \sigma_1 \sigma_2 - \sigma_2 \sigma_3 - \sigma_3 \sigma_1)}$$

$$= \sqrt{\boldsymbol{\sigma}^d : \boldsymbol{\sigma}^d}$$

# Principal stress space

$$\begin{aligned}\vec{\sigma} &= \vec{\sigma}^h + \vec{\sigma}^d = \sigma^h \vec{e}_p + \sigma^d \vec{e}_t(\phi) \\ &= \sigma^h \vec{e}_p + \sigma^d \{\cos(\phi) \vec{e}_q - \sin(\phi) \vec{e}_r\} \\ &= \sigma^h \frac{1}{3} \sqrt{3} (\vec{e}_1 + \vec{e}_2 + \vec{e}_3) + \sigma^d \left\{ \cos(\phi) \frac{1}{6} \sqrt{6} (2\vec{e}_1 - \vec{e}_2 - \vec{e}_3) - \sin(\phi) \frac{1}{2} \sqrt{2} (\vec{e}_2 - \vec{e}_3) \right\} \\ &= \left\{ \frac{1}{3} \sqrt{3} \sigma^h + \frac{1}{3} \sqrt{6} \sigma^d \cos(\phi) \right\} \vec{e}_1 + \\ &\quad \left\{ \frac{1}{3} \sqrt{3} \sigma^h - \frac{1}{6} \sqrt{6} \sigma^d \cos(\phi) - \frac{1}{2} \sqrt{2} \sigma^d \sin(\phi) \right\} \vec{e}_2 + \\ &\quad \left\{ \frac{1}{3} \sqrt{3} \sigma^h - \frac{1}{6} \sqrt{6} \sigma^d \cos(\phi) + \frac{1}{2} \sqrt{2} \sigma^d \sin(\phi) \right\} \vec{e}_3 \\ &= \sigma_1 \vec{e}_1 + \sigma_2 \vec{e}_2 + \sigma_3 \vec{e}_3\end{aligned}$$

# Maximum stress/strain

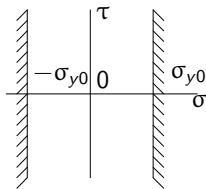
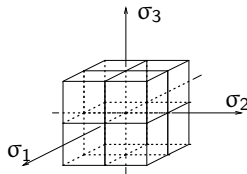
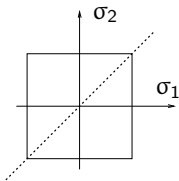
$$\sigma_{ij} = \sigma_{max} \quad | \quad \varepsilon_{ij} = \varepsilon_{max} \quad ; \quad \{i,j\} = \{1,2,3\} \quad (\text{orthotropic materials})$$

# Rankine

$$|\sigma_{max}| = \max(|\sigma_i| ; i = 1, 2, 3) = \sigma_{max,t} = \sigma_{y0} \quad (\text{brittle materials; cast iron})$$

# Rankine

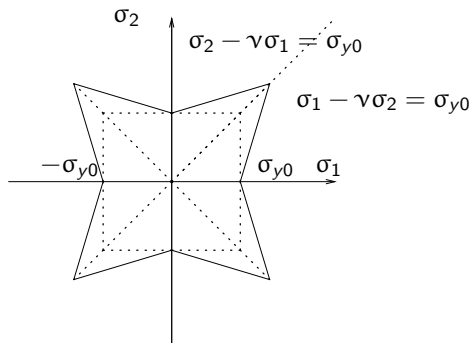
$$|\sigma_{max}| = \max(|\sigma_i| ; i = 1, 2, 3) = \sigma_{max,t} = \sigma_{y0} \quad (\text{brittle materials; cast iron})$$





# De Saint Venant

$$\varepsilon_{\max} = \max(|\varepsilon_i| ; i = 1, 2, 3) = \varepsilon_{\max_t} = \varepsilon_{y0} = \frac{\sigma_{y0}}{E} \quad (\text{brittle materials; cast iron})$$



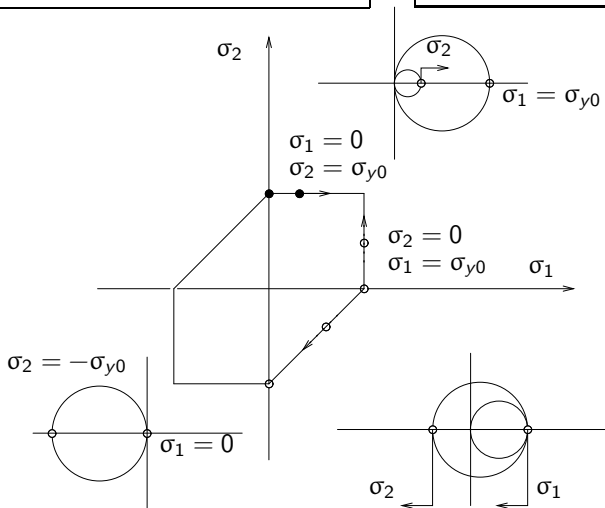
# Tresca

$$\tau_{max} = \frac{1}{2}(\sigma_{max} - \sigma_{min}) = \tau_{max,t} = \frac{1}{2}\sigma_{y0} \rightarrow \bar{\sigma}_{TR} = \sigma_{max} - \sigma_{min} = \sigma_{y0}$$

# Tresca : 2D yield contour

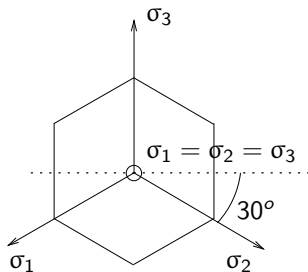
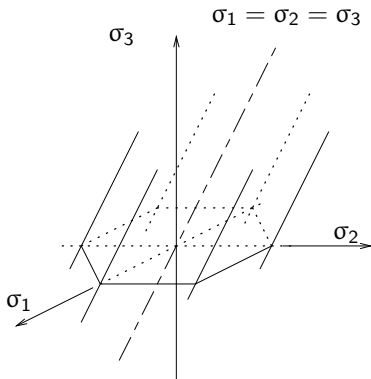
$$\tau_{max} = \frac{1}{2}(\sigma_{max} - \sigma_{min}) = \tau_{max,t} = \frac{1}{2}\sigma_{y0}$$

$$\bar{\sigma}_{TR} = \sigma_{max} - \sigma_{min} = \sigma_{y0}$$

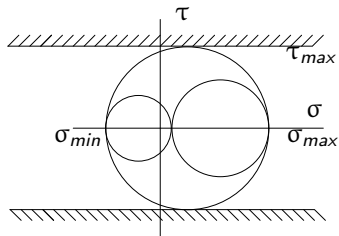


# Tresca : 3D yield surface

Mohr  $\rightarrow$  invariant for hydrostatic stress  $\rightarrow$   
yield surface  $//$  hydrostatic axis  
 $\Pi$  – plane  $\perp$  hydrostatic axis



# Tresca : st-plane



$$W^d = W_t^d$$

$$\begin{aligned} W^d &= \frac{1}{4G} \boldsymbol{\sigma}^d : \boldsymbol{\sigma}^d = \frac{1}{4G} \left\{ \boldsymbol{\sigma} : \boldsymbol{\sigma} - \frac{1}{3} \text{tr}^2(\boldsymbol{\sigma}) \right\} \quad \left( = -\frac{1}{2G} J_2(\boldsymbol{\sigma}^d) \right) \\ &= \frac{1}{4G} (\sigma_1^2 + \sigma_2^2 + \sigma_3^2) - \frac{1}{12G} (\sigma_1 + \sigma_2 + \sigma_3)^2 \\ &= \frac{1}{4G} \frac{1}{3} \{ (\sigma_1 - \sigma_2)^2 + (\sigma_2 - \sigma_3)^2 + (\sigma_3 - \sigma_1)^2 \} \end{aligned}$$

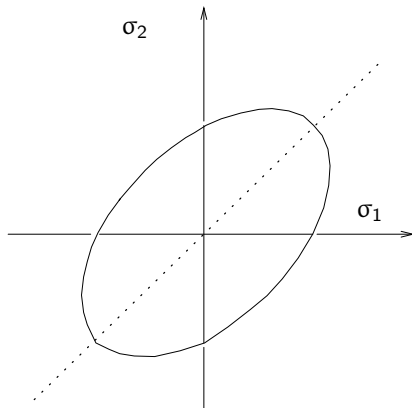
$$W_t^d = \frac{1}{4G} \frac{1}{3} \{ (\sigma - 0)^2 + (0 - 0)^2 + (0 - \sigma)^2 \} = \frac{1}{4G} \frac{1}{3} 2\sigma^2 = \frac{1}{4G} \frac{1}{3} 2\sigma_{y0}^2$$

$$\bar{\sigma}_{VM} = \sqrt{\frac{1}{2} \{ (\sigma_1 - \sigma_2)^2 + (\sigma_2 - \sigma_3)^2 + (\sigma_3 - \sigma_1)^2 \}} = \sigma_{y0}$$

## Von Mises : Cartesian stress components

$$\begin{aligned}\bar{\sigma}_{VM}^2 &= \frac{3}{2} \underline{\sigma}^d : \underline{\sigma}^d = 3J_2 \\ &= \frac{3}{2} \text{tr}(\underline{\sigma}^d \underline{\sigma}^d) \quad \text{with } \underline{\sigma}^d = \underline{\sigma} - \frac{1}{3} \text{tr}(\underline{\sigma}) \underline{I} \\ &= \frac{3}{2} \left\{ \left( \frac{2}{3} \sigma_{xx} - \frac{1}{3} \sigma_{yy} - \frac{1}{3} \sigma_{zz} \right)^2 + \sigma_{xy}^2 + \sigma_{xz}^2 + \right. \\ &\quad \left( \frac{2}{3} \sigma_{yy} - \frac{1}{3} \sigma_{zz} - \frac{1}{3} \sigma_{xx} \right)^2 + \sigma_{yz}^2 + \sigma_{yx}^2 + \\ &\quad \left. \left( \frac{2}{3} \sigma_{zz} - \frac{1}{3} \sigma_{xx} - \frac{1}{3} \sigma_{yy} \right)^2 + \sigma_{zx}^2 + \sigma_{zy}^2 \right\} \\ &= (\sigma_{xx}^2 + \sigma_{yy}^2 + \sigma_{zz}^2) - (\sigma_{xx} \sigma_{yy} + \sigma_{yy} \sigma_{zz} + \sigma_{zz} \sigma_{xx}) + 2 (\sigma_{xy}^2 + \sigma_{yz}^2 + \sigma_{zx}^2) \\ &= \sigma_{y0}^2\end{aligned}$$

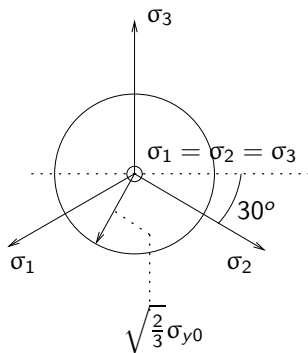
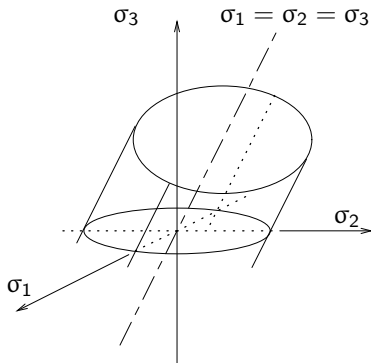
## Von Mises : 2D yield surface





# Von Mises : 3D yield surface

$$\frac{1}{2} \{ (\sigma_1 - \sigma_2)^2 + (\sigma_2 - \sigma_3)^2 + (\sigma_3 - \sigma_1)^2 \} = \sigma_{y0}^2$$



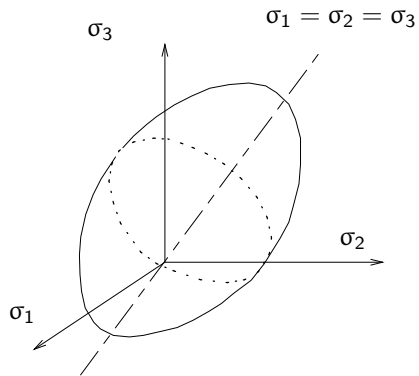
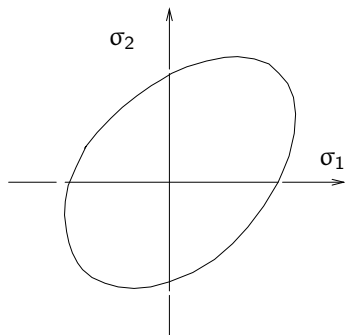
$$W = W_t$$

$$\begin{aligned} W &= W^h + W^d = \frac{1}{18K} \text{tr}^2(\boldsymbol{\sigma}) + \frac{1}{4G} \boldsymbol{\sigma}^d : \boldsymbol{\sigma}^d \\ &= \left( \frac{1}{18K} - \frac{1}{12G} \right) (\sigma_1 + \sigma_2 + \sigma_3)^2 + \frac{1}{4G} (\sigma_1^2 + \sigma_2^2 + \sigma_3^2) \end{aligned}$$

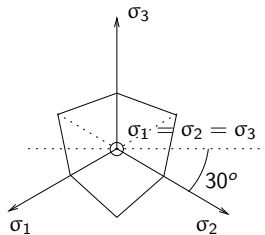
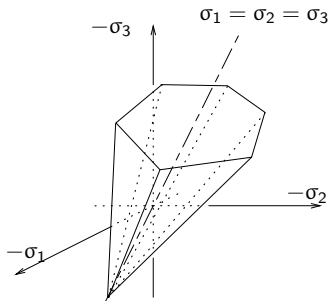
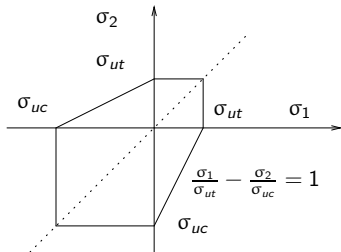
$$W_t = \left( \frac{1}{18K} - \frac{1}{12G} \right) \sigma^2 + \frac{1}{4G} \sigma^2 = \frac{1}{2E} \sigma^2 = \frac{1}{2E} \sigma_{y0}^2$$

$$2E \left( \frac{1}{18K} - \frac{1}{12G} \right) (\sigma_1 + \sigma_2 + \sigma_3)^2 + \frac{2E}{4G} (\sigma_1^2 + \sigma_2^2 + \sigma_3^2) = \sigma_{y0}^2$$

# Beltrami-Haigh : 2D/3D yield surface

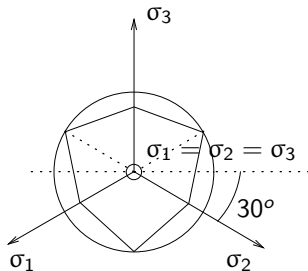
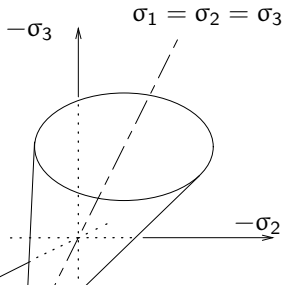
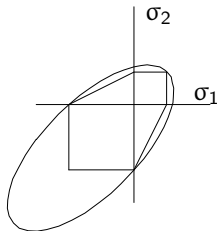


# Mohr-Coulomb : 2D/3D yield surface



# Drucker-Prager

$$\sqrt{\frac{2}{3} \boldsymbol{\sigma}^d : \boldsymbol{\sigma}^d} + \frac{6 \sin(\phi)}{3 - \sin(\phi)} p = \frac{6 \cos(\phi)}{3 - \sin(\phi)} C$$



## Other yield criteria

parabolic Drucker-Prager  $\left(3J_2 + \sqrt{3}\beta\sigma_{y0}J_1\right)^{\frac{1}{2}} = \sigma_{y0}$

Buyokozturk  $\left(3J_2 + \sqrt{3}\beta\sigma_{y0}J_1 - 0.2J_1^2\right)^{\frac{1}{2}} = \sigma_{y0}$

Hill  $\frac{\sigma_{11}^2}{X^2} - \frac{\sigma_{11}\sigma_{22}}{XY} + \frac{\sigma_{22}^2}{Y^2} + \frac{\sigma_{12}^2}{S^2}$

Hoffman

$$\left(\frac{1}{X_t} - \frac{1}{X_c}\right)\sigma_{11} + \left(\frac{1}{Y_t} - \frac{1}{Y_c}\right)\sigma_{22} + \left(\frac{1}{X_t X_c}\right)\sigma_{11}^2 + \left(\frac{1}{Y_t Y_c}\right)\sigma_{22}^2 + \left(\frac{1}{S^2}\right)\sigma_{12}^2 - \left(\frac{1}{X_t X_c}\right)\sigma_{11}\sigma_{22} = 0$$

Tsai-Wu

$$\left(\frac{1}{X_t} - \frac{1}{X_c}\right)\sigma_{11} + \left(\frac{1}{Y_t} - \frac{1}{Y_c}\right)\sigma_{22} + \left(\frac{1}{X_t X_c}\right)\sigma_{11}^2 + \left(\frac{1}{Y_t Y_c}\right)\sigma_{22}^2 + \left(\frac{1}{S^2}\right)\sigma_{12}^2 + 2F_{12}\sigma_{11}\sigma_{22} = 0$$

with  $F_{12}^2 > \frac{1}{X_t X_c} \frac{1}{Y_t Y_c}$

# GOVERNING EQUATIONS

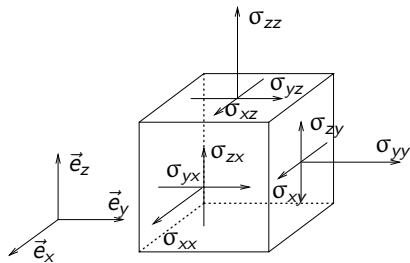
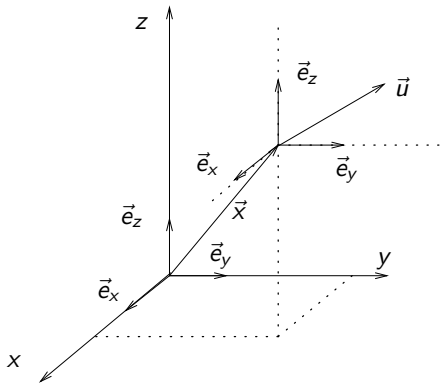
[back to index](#)

# Vector/tensor equations

gradient operator	:	$\vec{\nabla} = \underline{\nabla}^T \underline{\tilde{e}}$
position	:	$\vec{x} = \underline{x}^T \underline{\tilde{e}}$
displacement	:	$\vec{u} = \underline{u}^T \underline{\tilde{e}}$
strain	:	$\underline{\varepsilon} = \frac{1}{2} \left\{ \left( \vec{\nabla} \vec{u} \right)^T + \left( \vec{\nabla} \vec{u} \right) \right\} = \underline{\tilde{e}}^T \underline{\underline{\varepsilon}} \underline{\tilde{e}}$
compatibility	:	$\nabla^2 \{ \text{tr}(\underline{\varepsilon}) \} - \vec{\nabla} \cdot (\vec{\nabla} \cdot \underline{\varepsilon})^T = 0$
stress	:	$\underline{\sigma} = \underline{\tilde{e}}^T \underline{\underline{\sigma}} \underline{\tilde{e}}$
balance laws	:	$\vec{\nabla} \cdot \underline{\sigma}^T + \rho \vec{q} = \rho \ddot{\vec{u}} \quad ; \quad \underline{\sigma} = \underline{\sigma}^T$
material law	:	$\underline{\sigma} = {}^4\mathbf{C} : \underline{\varepsilon} \quad ; \quad \underline{\varepsilon} = {}^4\mathbf{C}^{-1} : \underline{\sigma} = {}^4\mathbf{S} : \underline{\sigma}$
th.mech. mat. law	:	$\underline{\sigma} = {}^4\mathbf{C} : (\underline{\varepsilon} - \alpha \Delta T \mathbf{I}) \quad : \quad \underline{\varepsilon} = {}^4\mathbf{S} : \underline{\sigma} + \alpha \Delta T \mathbf{I}$



# Cartesian components



## Cartesian components, 3D

$$\underline{\tilde{x}}^T = \begin{bmatrix} x & y & z \end{bmatrix} \quad ; \quad \underline{\tilde{\nabla}}^T = \begin{bmatrix} \frac{\partial}{\partial x} & \frac{\partial}{\partial y} & \frac{\partial}{\partial z} \end{bmatrix} \quad ; \quad \underline{\tilde{u}}^T = \begin{bmatrix} u_x & u_y & u_z \end{bmatrix}$$

$$\underline{\tilde{\varepsilon}} = \frac{1}{2} \begin{bmatrix} 2u_{x,x} & u_{x,y} + u_{y,x} & u_{x,z} + u_{z,x} \\ \cdots & 2u_{y,y} & u_{y,z} + u_{z,y} \\ \cdots & \cdots & 2u_{z,z} \end{bmatrix}$$

$$2\varepsilon_{xy,xy} - \varepsilon_{xx,yy} - \varepsilon_{yy,xx} = 0 \quad \rightarrow \quad \text{cyc. } 2x$$

$$\varepsilon_{xx,yz} + \varepsilon_{yz,xx} - \varepsilon_{zx,xy} - \varepsilon_{xy,xz} = 0 \quad \rightarrow \quad \text{cyc. } 2x$$

$$\underline{\tilde{\varepsilon}}^T = \underline{\tilde{\varepsilon}}^T = \begin{bmatrix} \varepsilon_{xx} & \varepsilon_{yy} & \varepsilon_{zz} & \varepsilon_{xy} & \varepsilon_{yz} & \varepsilon_{zx} \end{bmatrix}$$

$$\underline{\tilde{\sigma}}^T = \underline{\tilde{\sigma}}^T = \begin{bmatrix} \sigma_{xx} & \sigma_{yy} & \sigma_{zz} & \sigma_{xy} & \sigma_{yz} & \sigma_{zx} \end{bmatrix}$$

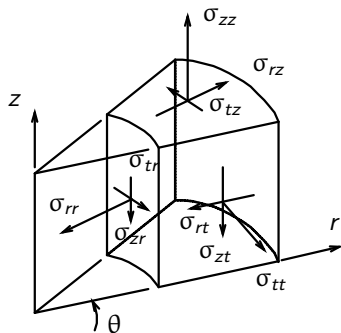
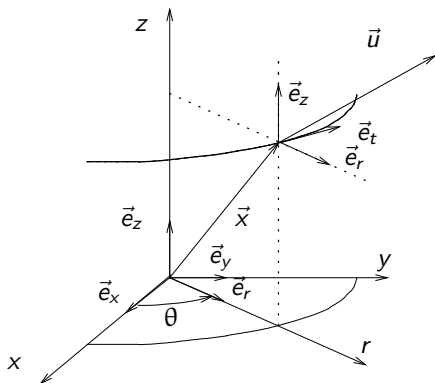
$$\sigma_{xx,x} + \sigma_{xy,y} + \sigma_{xz,z} + \rho q_x = \rho \ddot{u}_x \quad (\sigma_{xy} = \sigma_{yx})$$

$$\sigma_{yx,x} + \sigma_{yy,y} + \sigma_{yz,z} + \rho q_y = \rho \ddot{u}_y \quad (\sigma_{yz} = \sigma_{zy})$$

$$\sigma_{zx,x} + \sigma_{zy,y} + \sigma_{zz,z} + \rho q_z = \rho \ddot{u}_z \quad (\sigma_{zx} = \sigma_{xz})$$

$$\underline{\tilde{\sigma}} = \underline{\underline{C}} \underline{\tilde{\varepsilon}} \underline{\tilde{\sigma}} = \underline{\underline{C}} \underline{\tilde{\varepsilon}} \quad ; \quad \underline{\tilde{\varepsilon}} = \underline{\underline{S}} \underline{\tilde{\sigma}} \underline{\tilde{\varepsilon}} = \underline{\underline{S}} \underline{\tilde{\sigma}}$$

# Cylindrical components



## Cylindrical components, 3D

$$\underline{x}^T = \begin{bmatrix} r & \theta & z \end{bmatrix} \quad ; \quad \underline{\nabla}^T = \begin{bmatrix} \frac{\partial}{\partial r} & \frac{1}{r} \frac{\partial}{\partial \theta} & \frac{\partial}{\partial z} \end{bmatrix} \quad ; \quad \underline{u}^T = \begin{bmatrix} u_r & u_t & u_z \end{bmatrix}$$

$$\underline{\varepsilon} = \frac{1}{2} \begin{bmatrix} 2u_{r,r} & \frac{1}{r}(u_{r,t} - u_t) + u_{t,r} & u_{r,z} + u_{z,r} \\ \cdots & 2\frac{1}{r}(u_r + u_{t,t}) & \frac{1}{r}u_{z,t} + u_{t,z} \\ \cdots & \cdots & 2u_{z,z} \end{bmatrix}$$

$$2\varepsilon_{rt,rt} - \varepsilon_{rr,tt} - \varepsilon_{tt,rr} = 0 \quad \rightarrow \quad \text{cyc. 2x}$$

$$\varepsilon_{rr,tz} + \varepsilon_{tz,rr} - \varepsilon_{zr,rt} - \varepsilon_{rt,rz} = 0 \quad \rightarrow \quad \text{cyc. 2x}$$

$$\begin{aligned} \underline{\underline{\varepsilon}}^T &= \underline{\underline{\varepsilon}}^T = \begin{bmatrix} \varepsilon_{rr} & \varepsilon_{tt} & \varepsilon_{zz} & \varepsilon_{rt} & \varepsilon_{tz} & \varepsilon_{zr} \end{bmatrix} \\ \underline{\underline{\sigma}}^T &= \underline{\underline{\sigma}}^T = \begin{bmatrix} \sigma_{rr} & \sigma_{tt} & \sigma_{zz} & \sigma_{rt} & \sigma_{tz} & \sigma_{zr} \end{bmatrix} \end{aligned}$$

$$\sigma_{rr,r} + \frac{1}{r} \sigma_{rt,t} + \frac{1}{r} (\sigma_{rr} - \sigma_{tt}) + \sigma_{rz,z} + \rho q_r = \rho \ddot{u}_r \quad (\sigma_{rt} = \sigma_{tr})$$

$$\sigma_{tr,r} + \frac{1}{r} \sigma_{tt,t} + \frac{1}{r} (\sigma_{tr} + \sigma_{rt}) + \sigma_{tz,z} + \rho q_t = \rho \ddot{u}_t \quad (\sigma_{tz} = \sigma_{zt})$$

$$\sigma_{zr,r} + \frac{1}{r} \sigma_{rt,t} + \frac{1}{r} \sigma_{zr} + \sigma_{rz,z} + \rho q_z = \rho \ddot{u}_z \quad (\sigma_{zr} = \sigma_{rz})$$

$$\underline{\underline{\sigma}} = \underline{\underline{C}}_{\underline{\underline{\varepsilon}}} \underline{\underline{\sigma}} = \underline{\underline{C}}_{\underline{\underline{\varepsilon}}} \quad ; \quad \underline{\underline{\varepsilon}} = \underline{\underline{S}}_{\underline{\underline{\sigma}}} \underline{\underline{\varepsilon}} = \underline{\underline{S}}_{\underline{\underline{\sigma}}} \underline{\underline{\sigma}}$$

# Material law, 3D (No $\Delta T$ )

$$\underline{\underline{C}} = \begin{bmatrix} A & Q & R & 0 & 0 & 0 \\ Q & B & S & 0 & 0 & 0 \\ R & S & C & 0 & 0 & 0 \\ 0 & 0 & 0 & 2K & 0 & 0 \\ 0 & 0 & 0 & 0 & 2L & 0 \\ 0 & 0 & 0 & 0 & 0 & 2M \end{bmatrix} \rightarrow \underline{\underline{S}} = \underline{\underline{C}}^{-1} = \begin{bmatrix} a & q & r & 0 & 0 & 0 \\ q & b & s & 0 & 0 & 0 \\ r & s & c & 0 & 0 & 0 \\ 0 & 0 & 0 & \frac{1}{2}k & 0 & 0 \\ 0 & 0 & 0 & 0 & \frac{1}{2}l & 0 \\ 0 & 0 & 0 & 0 & 0 & \frac{1}{2}m \end{bmatrix}$$

quadratic

$$B = A ; S = R ; M = L ;$$

transversal isotropic

$$B = A ; S = R ; M = L ; K = \frac{1}{2}(A - Q)$$

cubic

$$C = B = A ; S = R = Q ; M = L = K \neq \frac{1}{2}(A - Q)$$

isotropic

$$C = B = A ; S = R = Q ; M = L = K = \frac{1}{2}(A - Q)$$

3D isotropic



3D transversal isotropic



3D orthotropic



3D general orthotropic



plane strain / plane stress / planar



# Planar deformation : Cartesian

$$\left. \begin{array}{ll} \text{plane strain} & : \quad \varepsilon_{zz} = \varepsilon_{xz} = \varepsilon_{yz} = 0 \\ \text{plane stress} & : \quad \sigma_{zz} = \sigma_{xz} = \sigma_{yz} = 0 \end{array} \right\} \quad \left\{ \begin{array}{l} u_x = u_x(x, y) \\ u_y = u_y(x, y) \end{array} \right.$$

$$\underline{\underline{\varepsilon}}^T = \underline{\underline{\varepsilon}} = \begin{bmatrix} \varepsilon_{xx} & \varepsilon_{yy} & \varepsilon_{xy} \end{bmatrix} = \begin{bmatrix} u_{x,x} & u_{y,y} & \frac{1}{2}(u_{x,y} + u_{y,x}) \end{bmatrix}$$

$$2\varepsilon_{xy,xy} - \varepsilon_{xx,yy} - \varepsilon_{yy,xx} = 0$$

$$\underline{\underline{\sigma}}^T = \underline{\underline{\sigma}} = \begin{bmatrix} \sigma_{xx} & \sigma_{yy} & \sigma_{xy} \end{bmatrix}$$

$$\sigma_{xx,x} + \sigma_{xy,y} + \rho q_x = \rho \ddot{u}_x \quad (\sigma_{xy} = \sigma_{yx})$$

$$\sigma_{yx,x} + \sigma_{yy,y} + \rho q_y = \rho \ddot{u}_y$$

$$\underline{\underline{C}}_p = \begin{bmatrix} A_p & Q_p & 0 \\ Q_p & B_p & 0 \\ 0 & 0 & 2K \end{bmatrix} \quad ; \quad \underline{\underline{S}}_p = \begin{bmatrix} a_p & q_p & 0 \\ q_p & b_p & 0 \\ 0 & 0 & \frac{1}{2}k \end{bmatrix}$$

orthotr. pe/ps ▷▷

transv.iso. pe/ps ▷▷

isotropic pe/ps ▷▷

# Planar deformation : cylindrical

$$\left. \begin{array}{l} \text{plane strain} \\ \text{plane stress} \end{array} \right\} : \left. \begin{array}{l} \varepsilon_{zz} = \varepsilon_{rz} = \varepsilon_{tz} = 0 \\ \sigma_{zz} = \sigma_{rz} = \sigma_{tz} = 0 \end{array} \right\} \quad \left\{ \begin{array}{l} u_r = u_r(r, \theta) \\ u_t = u_t(r, \theta) \end{array} \right.$$

$$\underline{\underline{\varepsilon}}^T = \underline{\underline{\varepsilon}} = \begin{bmatrix} \varepsilon_{rr} & \varepsilon_{tt} & \varepsilon_{rt} \end{bmatrix} = \begin{bmatrix} u_{r,r} & \frac{1}{r}(u_r + u_{t,t}) & \frac{1}{2} \left( \frac{1}{r}(u_{r,t} - u_t) + u_{t,r} \right) \end{bmatrix}$$

$$2\varepsilon_{rt,rt} - \varepsilon_{rr,tt} - \varepsilon_{tt,rr} = 0$$

$$\underline{\underline{\sigma}}^T = \underline{\underline{\sigma}} = \begin{bmatrix} \sigma_{rr} & \sigma_{tt} & \sigma_{rt} \end{bmatrix}$$

$$\sigma_{rr,r} + \frac{1}{r} \sigma_{rt,t} + \frac{1}{r} (\sigma_{rr} - \sigma_{tt}) + \rho q_r = \rho \ddot{u}_r \quad (\sigma_{rt} = \sigma_{tr})$$

$$\sigma_{tr,r} + \frac{1}{r} \sigma_{tt,t} + \frac{1}{r} (\sigma_{tr} + \sigma_{rt}) + \rho q_t = \rho \ddot{u}_t$$

$$\underline{\underline{C}}_p = \begin{bmatrix} A_p & Q_p & 0 \\ Q_p & B_p & 0 \\ 0 & 0 & 2K \end{bmatrix} \quad ; \quad \underline{\underline{S}}_p = \begin{bmatrix} a_p & q_p & 0 \\ q_p & b_p & 0 \\ 0 & 0 & \frac{1}{2}k \end{bmatrix}$$

orthotr. pe/ps ▷ ▷

transv.iso. pe/ps ▷ ▷

isotropic pe/ps ▷ ▷

# Axi-symmetric + $u_t = 0$

$$\left. \begin{array}{ll} \text{plane strain} & : \quad \varepsilon_{zz} = \varepsilon_{rz} = \varepsilon_{tz} = 0 \\ \text{plane stress} & : \quad \sigma_{zz} = \sigma_{rz} = \sigma_{tz} = 0 \end{array} \right\} \quad \left\{ \begin{array}{l} u_r = u_r(r) \\ u_t = 0 \end{array} \right.$$

$$\underline{\underline{\varepsilon}}^T = \underline{\varepsilon}^T = \begin{bmatrix} \varepsilon_{rr} & \varepsilon_{tt} \end{bmatrix} = \begin{bmatrix} u_{r,r} & \frac{1}{r}(u_r) \end{bmatrix}$$

$$\varepsilon_{rr} = u_{r,r} = (r\varepsilon_{tt})_{,r} = \varepsilon_{tt} + r\varepsilon_{tt,r}$$

$$\underline{\underline{\sigma}}^T = \underline{\sigma}^T = \begin{bmatrix} \sigma_{rr} & \sigma_{tt} \end{bmatrix}$$

$$\sigma_{rr,r} + \frac{1}{r}(\sigma_{rr} - \sigma_{tt}) + \rho q_r = \rho \ddot{u}_r$$

$$\underline{\underline{C}}_p = \begin{bmatrix} A_p & Q_p \\ Q_p & B_p \end{bmatrix} \quad ; \quad \underline{\underline{S}}_p = \begin{bmatrix} a_p & q_p \\ q_p & b_p \end{bmatrix}$$

orthotr. pe/ps ▷ ▷

transv.iso. pe/ps ▷ ▷

isotropic pe/ps ▷ ▷



## SOLUTION STRATEGIES

[back to index](#)

# Governing equations

## unknown variables

$$\begin{array}{ll} \text{displacements} & : \quad \vec{u} = \vec{u}(\vec{x}) \quad \rightarrow \quad \mathbf{F} = \left( \vec{\nabla}_0 \vec{x} \right)^T \quad \rightarrow \quad \mathbf{E}, \boldsymbol{\varepsilon} \\ \text{stresses} & : \quad \boldsymbol{\sigma} \quad \rightarrow \quad g(\boldsymbol{\sigma}) = g(\sigma_1, \sigma_2, \sigma_3) = g_t \end{array}$$

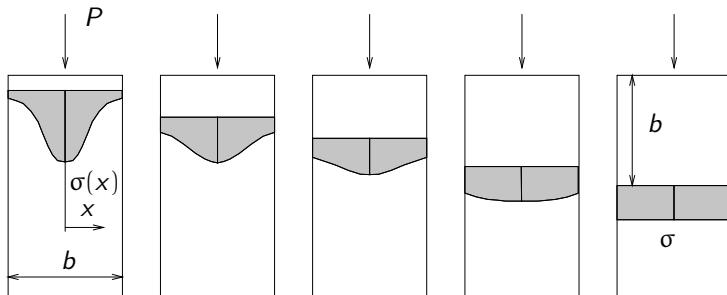
## equations

$$\begin{array}{ll} \text{compatibility} & : \quad \nabla^2 \{ \text{tr}(\boldsymbol{\varepsilon}) \} - \vec{\nabla} \cdot (\vec{\nabla} \cdot \boldsymbol{\varepsilon})^T = 0 \\ \text{equilibrium} & : \quad \vec{\nabla} \cdot \boldsymbol{\sigma}^T + \rho \vec{q} = \rho \ddot{\vec{u}} \quad ; \quad \boldsymbol{\sigma} = \boldsymbol{\sigma}^T \\ \text{material law} & : \quad \boldsymbol{\sigma} = \boldsymbol{\sigma}(\mathbf{F}) \quad \rightarrow \quad \boldsymbol{\sigma} = {}^4\mathbf{C} : \boldsymbol{\varepsilon} \quad \rightarrow \quad \boldsymbol{\varepsilon} = {}^4\mathbf{S} : \boldsymbol{\sigma} \end{array}$$

## boundary conditions

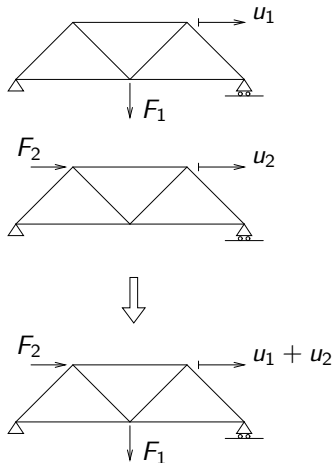
$$\begin{array}{ll} \text{displacement} & : \quad \vec{u} = \vec{u}_p \quad \forall \quad \vec{x} \in A_u \\ \text{edge load} & : \quad \vec{p} = \vec{n} \cdot \boldsymbol{\sigma} = \vec{p}_p \quad \forall \quad \vec{x} \in A_p \end{array}$$

# Saint-Venant's principle



$$P = \int_A \sigma(x) dA = \sigma A \quad ; \quad A = b * t$$

# Superposition



## Solution : displacement method

$$\left. \begin{array}{l} \vec{\nabla} \cdot \boldsymbol{\sigma}^T + \rho \vec{q} = \rho \ddot{\vec{u}} \\ \boldsymbol{\sigma} = {}^4\mathbf{C} : \boldsymbol{\varepsilon} \end{array} \right\} \rightarrow \left. \begin{array}{l} \vec{\nabla} \cdot ({}^4\mathbf{C} : \boldsymbol{\varepsilon})^T + \rho \vec{q} = \rho \ddot{\vec{u}} \\ \boldsymbol{\varepsilon} = \frac{1}{2} \left\{ \left( \vec{\nabla} \vec{u} \right)^T + \left( \vec{\nabla} \vec{u} \right) \right\} \end{array} \right\} \rightarrow$$

$$\vec{\nabla} \cdot \left\{ {}^4\mathbf{C} : \left( \vec{\nabla} \vec{u} \right) \right\}^T + \rho \vec{q} = \rho \ddot{\vec{u}} \quad \rightarrow$$

$$\vec{u} \rightarrow \boldsymbol{\varepsilon} \rightarrow \boldsymbol{\sigma}$$

# Planar, Cartesian : Navier equations

$$\left. \begin{aligned} \sigma_{xx,x} + \sigma_{xy,y} + \rho q_x &= \rho \ddot{u}_x & ; & & \sigma_{yx,x} + \sigma_{yy,y} + \rho q_y &= \rho \ddot{u}_y \\ \sigma_{xx} &= A_p \varepsilon_{xx} + Q_p \varepsilon_{yy} \\ \sigma_{yy} &= Q_p \varepsilon_{xx} + B_p \varepsilon_{yy} \\ \sigma_{xy} &= 2K \varepsilon_{xy} \end{aligned} \right\}$$

$$\left. \begin{aligned} A_p \varepsilon_{xx,x} + Q_p \varepsilon_{yy,x} + 2K \varepsilon_{xy,y} + \rho q_x &= \rho \ddot{u}_x \\ 2K \varepsilon_{xy,x} + Q_p \varepsilon_{xx,y} + B_p \varepsilon_{yy,y} + \rho q_y &= \rho \ddot{u}_y \end{aligned} \right\}$$

$$\left. \begin{aligned} A_p u_{x,xx} + K u_{x,yy} + (Q_p + K) u_{y,yx} + \rho q_x &= \rho \ddot{u}_x \\ K u_{y,xx} + B_p u_{y,yy} + (Q_p + K) u_{x,xy} + \rho q_y &= \rho \ddot{u}_y \end{aligned} \right\}$$

Planar, axi-symmetric with  $u_t = 0$ , isotropic

$$\left. \begin{aligned}
 \sigma_{rr,r} + \frac{1}{r}(\sigma_{rr} - \sigma_{tt}) + \rho q_r &= \rho \ddot{u}_r \\
 \sigma_{rr} &= A_p \varepsilon_{rr} + Q_p \varepsilon_{tt} - \Theta_{p1} \alpha \Delta T \\
 \sigma_{tt} &= Q_p \varepsilon_{rr} + A_p \varepsilon_{tt} - \Theta_{p1} \alpha \Delta T
 \end{aligned} \right\}$$

$$\left. \begin{aligned}
 A_p \varepsilon_{rr,r} + Q_p \varepsilon_{tt,r} - \Theta_{p1} \alpha (\Delta T)_{,r} + \\
 \frac{1}{r} \{ (A_p - Q_p) \varepsilon_{rr} + (Q_p - A_p) \varepsilon_{tt} \} + \rho q_r &= \rho \ddot{u}_r \\
 \varepsilon_{rr} = u_{r,r} \quad ; \quad \varepsilon_{tt} = \frac{1}{r} u_r
 \end{aligned} \right\}$$

$$u_{r,rr} + \frac{1}{r} u_{r,r} - \frac{1}{r^2} u_r = \frac{\rho}{A_p} (\ddot{u}_r - q_r) + \frac{\Theta_{p1}}{A_p} \alpha (\Delta T)_{,r} = f(r)$$

# WEIGHTED RESIDUAL FORMULATION

[back to index](#)



# Weighted residual formulation for 3D deformation

equilibrium equation  $\vec{\nabla} \cdot \boldsymbol{\sigma}^T + \rho \vec{q} = \vec{0} \quad \forall \vec{x} \in V$

approximation  $\rightarrow$  residual  $\vec{\nabla} \cdot \boldsymbol{\sigma}^T + \rho \vec{q} = \vec{\Delta}(\vec{x}) \neq \vec{0} \quad \forall \vec{x} \in V$

weighted residual

$$\int_V \vec{w}(\vec{x}) \cdot \vec{\Delta}(\vec{x}) dV = \int_V \vec{w} \cdot [\vec{\nabla} \cdot \boldsymbol{\sigma}^T + \rho \vec{q}] dV$$

equivalent problem formulation

$$\int_V \vec{w} \cdot [\vec{\nabla} \cdot \boldsymbol{\sigma}^T + \rho \vec{q}] dV = 0 \quad \forall \vec{w}(\vec{x}) \quad \Leftrightarrow \quad \vec{\nabla} \cdot \boldsymbol{\sigma}^T + \rho \vec{q} = \vec{0} \quad \forall \vec{x} \in V$$

# Weak formulation

$$\left. \begin{aligned} \int_V \vec{w} \cdot \left[ \vec{\nabla} \cdot \boldsymbol{\sigma}^T + \rho \vec{q} \right] dV &= 0 \\ \vec{\nabla} \cdot (\boldsymbol{\sigma}^T \cdot \vec{w}) &= (\vec{\nabla} \vec{w})^T : \boldsymbol{\sigma}^T + \vec{w} \cdot (\vec{\nabla} \cdot \boldsymbol{\sigma}^T) \end{aligned} \right\} \rightarrow$$

$$\left. \begin{aligned} \int_V \left[ \vec{\nabla} \cdot (\boldsymbol{\sigma}^T \cdot \vec{w}) - (\vec{\nabla} \vec{w})^T : \boldsymbol{\sigma}^T + \vec{w} \cdot \rho \vec{q} \right] dV &= 0 \quad \forall \vec{w} \\ \text{Gauss / Stokes : } \int_V \vec{\nabla} \cdot (\boldsymbol{\sigma}^T \cdot \vec{w}) &= \int_V \vec{n} \cdot \boldsymbol{\sigma}^T \cdot \vec{w} dA = \int_A \vec{w} \cdot \vec{p} dA \end{aligned} \right\} \rightarrow$$

$$\int_V (\vec{\nabla} \vec{w})^T : \boldsymbol{\sigma} dV = \int_V \vec{w} \cdot \rho \vec{q} dV + \int_A \vec{w} \cdot \vec{p} dA \quad \forall \vec{w}$$

$$\int_V (\vec{\nabla} \vec{w})^T : \boldsymbol{\sigma} dV = f_e(\vec{w}) \quad \forall \vec{w}$$

# Linear elastic formulation

$$\int_{V_0} (\vec{\nabla} \vec{w})^T : \boldsymbol{\sigma} dV_0 = \int_{V_0} \vec{w} \cdot \rho \vec{q} dV_0 + \int_{A_0} \vec{w} \cdot \vec{p} dA_0 = f_{e0}(\vec{w}) \quad \forall \vec{w}$$

$$\begin{aligned} \boldsymbol{\sigma} &= {}^4\mathbf{C} : \boldsymbol{\varepsilon} \\ &= {}^4\mathbf{C} : \frac{1}{2} \left\{ (\vec{\nabla}_0 \vec{u}) + (\vec{\nabla}_0 \vec{u})^T \right\} = {}^4\mathbf{C} : (\vec{\nabla}_0 \vec{u}) \end{aligned}$$

$$\int_{V_0} (\vec{\nabla}_0 \vec{w})^T : {}^4\mathbf{C} : (\vec{\nabla}_0 \vec{u}) dV_0 = \int_{V_0} \vec{w} \cdot \rho \vec{q} dV_0 + \int_{A_0} \vec{w} \cdot \vec{p} dA_0 = f_{e0}(\vec{w}) \quad \forall \vec{w}$$

## Matrix/column notation

$$\int_{V_0} \left( \underline{L}_{0w} \right)_t^T \underline{C} \left( \underline{L}_{0u} \right)_t dV_0 = f_{e0}(\underline{w}) \quad \forall \underline{w}$$

# Total Lagrange formulation

$$\int_V (\vec{\nabla} \vec{w})^c : \boldsymbol{\sigma} dV = f_e(\vec{w}) \quad \forall \quad \vec{w}(\vec{x})$$

transformation to undeformed configuration  $t_0$

$$\begin{aligned} \vec{\nabla} &= \mathbf{F}^{-c} \cdot \vec{\nabla}_0 \quad \rightarrow \quad (\vec{\nabla} \vec{w})^c = (\vec{\nabla}_0 \vec{w})^c \cdot \mathbf{F}^{-1} \\ dV &= \det(\mathbf{F}) dV_0 = J dV_0 \end{aligned}$$

weighted residual integral

$$\left. \begin{aligned} \int_{V_0} (\vec{\nabla}_0 \vec{w})^c \cdot \mathbf{F}^{-1} : \boldsymbol{\sigma} J dV_0 &= f_{e0}(\vec{w}) \quad \forall \quad \vec{w}(\vec{x}) \\ \mathbf{P} &= J \mathbf{F}^{-1} \cdot \boldsymbol{\sigma} \cdot \mathbf{F}^{-c} \end{aligned} \right\} \rightarrow$$

$$\int_{V_0} (\vec{\nabla}_0 \vec{w})^c : (\mathbf{P} \cdot \mathbf{F}^c) dV_0 = f_{e0}(\vec{w}) \quad \forall \quad \vec{w}(\vec{x})$$

## Iterative solution process

$$\left. \begin{aligned} \int_{V_0} (\vec{\nabla}_0 \vec{w})^c : (\mathbf{P} \cdot \mathbf{F}^c) dV_0 &= f_{e0}(\vec{w}) \quad \forall \vec{w}(\vec{x}) \\ \mathbf{F} &= (\vec{\nabla}_0 \vec{x})^c = \{\vec{\nabla}_0(\vec{x}^* + \delta \vec{x})\}^c = (\vec{\nabla}_0 \vec{x}^*)^c + (\vec{\nabla}_0 \delta \vec{x})^c \\ &= \mathbf{F}^* + \delta \mathbf{F} = \mathbf{F}^* + \mathbf{L}_{0u} \\ \mathbf{P} &= \mathbf{P}^* + \delta \mathbf{P} \end{aligned} \right\} \rightarrow$$

$$\int_{V_0} (\vec{\nabla}_0 \vec{w})^c : (\mathbf{P}^* + \delta \mathbf{P}) \cdot (\mathbf{F}^* + \mathbf{L}_{0u})^c dV_0 = f_{e0}(\vec{w}) \quad \forall \vec{w}(\vec{x})$$

# Linearisation

$$\int_{V_0} \mathbf{L}_{0w} : (\mathbf{P}^* + \delta \mathbf{P}) \cdot (\mathbf{F}^* + \mathbf{L}_{0u})^c dV_0 = f_{e0}(\vec{w}) \quad \forall \quad \vec{w}(\vec{x})$$

$$\int_{V_0} \mathbf{L}_{0w} : (\mathbf{P}^* \cdot \mathbf{F}^{*c} + \mathbf{P}^* \cdot \mathbf{L}_{0u}^c + \delta \mathbf{P} \cdot \mathbf{F}^{*c}) dV_0 = f_{e0}(\vec{w}) \quad \forall \quad \vec{w}(\vec{x})$$

$$\int_{V_0} \mathbf{L}_{0w} : (\mathbf{P}^* \cdot \mathbf{L}_{0u}^c + \delta \mathbf{P} \cdot \mathbf{F}^{*c}) dV_0 =$$

$$f_{e0}(\vec{w}) - \int_{V_0} \mathbf{L}_{0w} : (\mathbf{P}^* \cdot \mathbf{F}^{*c}) dV_0 = r^* \quad \forall \quad \vec{w}(\vec{x})$$

# Material model

$$\delta \mathbf{P} = {}^4\mathbf{M} : \mathbf{L}_{0u} \rightarrow$$

$$\int_{V_0} \mathbf{L}_{0w} : (\mathbf{P}^* \cdot \mathbf{L}_{0u}^c + ({}^4\mathbf{M} : \mathbf{L}_{0u}) \cdot \mathbf{F}^{*c}) dV_0 =$$

$$f_{e0}(\vec{w}) - \int_{V_0} \mathbf{L}_{0w} : (\mathbf{P}^* \cdot \mathbf{F}^{*c}) dV_0 \quad \forall \quad \vec{w}(\vec{x})$$

$$\int_{V_0} \left[ \mathbf{L}_{0w} : (\mathbf{P}^* \cdot \mathbf{L}_{0u}^c) + \mathbf{L}_{0w} : (\mathbf{F}^* \cdot {}^4\mathbf{M}^{lrc}) : \mathbf{L}_{0u}^c \right] dV_0 =$$

$$f_{e0}(\vec{w}) - \int_{V_0} \mathbf{L}_{0w} : (\mathbf{P}^* \cdot \mathbf{F}^{*c}) dV_0 \quad \forall \quad \vec{w}(\vec{x})$$



## Matrix/column notation

$$\int_{V_0} \left[ \left( \underline{\tilde{L}}_{0w} \right)_t^T \underline{\underline{P}} \left( \underline{\tilde{L}}_{0u} \right)_t + \left( \underline{\tilde{L}}_{0w} \right)_t^T \underline{\underline{F}}_{cr} \underline{\underline{M}}_{0c} \left( \underline{\tilde{L}}_{0u} \right)_t \right] dV_0 =$$

$$f_{e0}(\underline{\tilde{w}}) - \int_{V_0} \left( \underline{\tilde{L}}_{0w} \right)_t^T \underline{\underline{F}}_{cr} \underline{\underline{P}} dV_0 = f_{e0}(\underline{\tilde{w}}) - f_{i0}(\underline{\tilde{w}})$$

$$\int_{V_0} \left( \underline{\tilde{L}}_{0w} \right)_t^T \left[ \underline{\underline{P}} + \underline{\underline{F}}_{cr} \underline{\underline{M}}_{0c} \right] \left( \underline{\tilde{L}}_{0u} \right)_t dV_0 = f_{e0}(\underline{\tilde{w}}) - f_{i0}(\underline{\tilde{w}})$$

# Updated Lagrange formulation

$$\int_V (\vec{\nabla} \vec{w})^c : \boldsymbol{\sigma} dV = f_e(\vec{w}) \quad \forall \quad \vec{w}(\vec{x})$$

transformation to begin increment configuration  $t_n$

$$\begin{aligned} \vec{\nabla} &= \mathbf{F}_n^{-c} \cdot \vec{\nabla}_n \quad \rightarrow \quad (\vec{\nabla} \vec{w})^c = (\vec{\nabla}_n \vec{w})^c \cdot \mathbf{F}_n^{-1} \\ dV &= \det(\mathbf{F}_n) dV_n \end{aligned}$$

weighted residual integral

$$\begin{aligned} \int_{V_n} (\vec{\nabla}_n \vec{w})^c \cdot \mathbf{F}_n^{-1} : \boldsymbol{\sigma} \det(\mathbf{F}_n) dV_n &= f_{en}(\vec{w}) \quad \forall \quad \vec{w}(\vec{x}) \quad \rightarrow \\ \int_{V_n} (\vec{\nabla}_n \vec{w})^c : (\mathbf{F}_n^{-1} \cdot \boldsymbol{\sigma}) \det(\mathbf{F}_n) dV_n &= f_{en}(\vec{w}) \quad \forall \quad \vec{w}(\vec{x}) \end{aligned}$$

# Iterative solution process

$$\left. \begin{aligned}
 \int_{V_n} (\vec{\nabla}_n \vec{w})^c : (\mathbf{F}_n^{-1} \cdot \boldsymbol{\sigma}) \det(\mathbf{F}_n) dV_n &= f_{en}(\vec{w}) \quad \forall \vec{w}(\vec{x}) \\
 \mathbf{F}_n &= (\vec{\nabla}_n \vec{x})^c = \{\vec{\nabla}_n(\vec{x}^* + \delta \vec{x})\}^c = (\vec{\nabla}_n \vec{x}^*)^c + (\vec{\nabla}_n \delta \vec{x})^c \\
 &= \mathbf{F}_n^* + \delta \mathbf{F}_n = \mathbf{F}_n^* + (\vec{\nabla}^* \delta \vec{x})^c \cdot (\vec{\nabla}_n \vec{x}^*)^c = \mathbf{F}_n^* + \mathbf{L}_u^* \cdot \mathbf{F}_n^* = (\mathbf{I} + \mathbf{L}_u^*) \cdot \mathbf{F}_n^* \\
 \boldsymbol{\sigma} &= \boldsymbol{\sigma}^* + \delta \boldsymbol{\sigma}
 \end{aligned} \right\} \rightarrow$$

$$\begin{aligned}
 \int_{V_n} (\vec{\nabla}_n \vec{w})^c : [(\mathbf{F}_n^*)^{-1} \cdot (\mathbf{I} + \mathbf{L}_u^*)^{-1} \cdot (\boldsymbol{\sigma}^* + \delta \boldsymbol{\sigma}) \det[(\mathbf{I} + \mathbf{L}_u^*) \cdot \mathbf{F}_n^*]] dV_n \\
 = f_{en}(\vec{w}) \quad \forall \vec{w}(\vec{x})
 \end{aligned}$$

# Linearisation

$$(\mathbf{I} + \mathbf{L}_u^*)^{-1} \approx \mathbf{I} - \mathbf{L}_u^*$$

$$\det[(\mathbf{I} + \mathbf{L}_u^*) \cdot \mathbf{F}_n^*] = \det(\mathbf{I} + \mathbf{L}_u^*) \det(\mathbf{F}_n^*) \approx \text{tr}(\mathbf{I} + \mathbf{L}_u^*) \det(\mathbf{F}_n^*) = (1 + \mathbf{I} : \mathbf{L}_u^*) \det(\mathbf{F}_n^*)$$

weighted residual integral

$$\begin{aligned} \int_{V_n} (\vec{\nabla}_n \vec{w})^c : \\ \left[ (\mathbf{F}_n^*)^{-1} \cdot (\mathbf{I} - \mathbf{L}_u^*) \cdot (\boldsymbol{\sigma}^* + \delta \boldsymbol{\sigma}) (1 + \mathbf{I} : \mathbf{L}_u^*) \det(\mathbf{F}_n^*) \right] dV_n \\ = f_{en}(\vec{w}) \quad \forall \quad \vec{w}(\vec{x}) \end{aligned}$$

further linearisation

$$\begin{aligned} \int_{V^*} [\mathbf{L}_w^* : \boldsymbol{\sigma}^* \mathbf{I} : \mathbf{L}_u^{*c} + \mathbf{L}_w^* : \delta \boldsymbol{\sigma} - \mathbf{L}_w^* : (\boldsymbol{\sigma}^{*c} \cdot \mathbf{L}_u^{*c})^c] dV^* = \\ f_e^*(\vec{w}) - \int_{V^*} \mathbf{L}_w^* : \boldsymbol{\sigma}^* dV^* = \\ r^* \quad \forall \quad \vec{w}(\vec{x}) \end{aligned}$$

# Material model

$$\delta \sigma = {}^4\mathbf{M} : \mathbf{L}_u^* \rightarrow$$

$$\int_{V^*} \left[ \mathbf{L}_w^* : \sigma^* \mathbf{I} : \mathbf{L}_u^{*c} + \mathbf{L}_w^* : {}^4\mathbf{M} : \mathbf{L}_u^* - \mathbf{L}_w^* : (\sigma^{*c} \cdot \mathbf{L}_u^{*c})^c \right] dV^* =$$

$$f_e^*(\vec{w}) - \int_{V^*} \mathbf{L}_w^* : \sigma^* dV^* \quad \forall \quad \vec{w}(\vec{x})$$

## Matrix/column notation

$$\int_{V^*} \left[ \left( \underline{\underline{L}}_w \right)_t^T \underline{\underline{\sigma}}^T \left( \underline{\underline{L}}_u \right)_t + \left( \underline{\underline{L}}_w \right)_t^T \underline{\underline{M}} \left( \underline{\underline{L}}_u \right)_t - \left( \underline{\underline{L}}_w \right)_t^T \underline{\underline{\sigma}}_{tr} \left( \underline{\underline{L}}_u \right)_t \right] dV^* =$$

$$f_e(\underline{\underline{w}}) - \int_{V^*} \left( \underline{\underline{L}}_w \right)_t^T \underline{\underline{\sigma}} dV^* =$$

$$f_e(\underline{\underline{w}}) - f_i(\underline{\underline{w}})$$

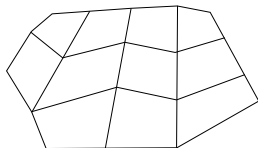
$$\int_{V^*} \left( \underline{\underline{L}}_w \right)_t^T \left[ \underline{\underline{\sigma}}^T - \underline{\underline{\sigma}}_{tr} + \underline{\underline{M}} \right] \left( \underline{\underline{L}}_u \right)_t dV^* = f_e(\underline{\underline{w}}) - f_i(\underline{\underline{w}})$$

$$\int_{V^*} \left( \underline{\underline{L}}_w \right)_t^T \left[ \underline{\underline{\Sigma}} + \underline{\underline{M}} \right] \left( \underline{\underline{L}}_u \right)_t dV^* = f_e(\underline{\underline{w}}) - f_i(\underline{\underline{w}})$$

# FINITE ELEMENT METHOD

[back to index](#)

# Discretisation

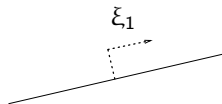
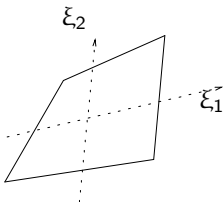
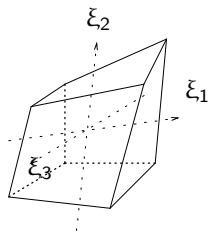


$$\int_V \left( \underline{\underline{L}}_w \right)_t^T \left[ \underline{\underline{W}} \right] \left( \underline{\underline{L}}_u \right)_t dV = f_e(\underline{\underline{w}}) - f_i(\underline{\underline{w}})$$

$$\sum_e \int_{V^e} \left( \underline{\underline{L}}_w \right)_t^T \left[ \underline{\underline{W}} \right] \left( \underline{\underline{L}}_u \right)_t dV^e = \sum_e f_e^e(\underline{\underline{w}}) - \sum_e f_i^e(\underline{\underline{w}}) \quad \forall \underline{\underline{w}}$$



# Isoparametric elements



isoparametric (local) coordinates

$$(\xi_1, \xi_2, \xi_3) \quad ; \quad -1 \leq \xi_i \leq 1 \quad i = 1, 2, 3$$

Jacobian matrix

$$\underline{J} = (\nabla_{\xi} x^T)^T dV^e = \det(\underline{J}) d\xi_1 d\xi_2 d\xi_3$$

# Interpolation

$$\begin{aligned}\vec{a} &= N^1 \vec{a}^1 + N^2 \vec{a}^2 + \dots + N^{nep} \vec{a}^{nep} = \tilde{N}^T \tilde{\vec{a}}^e \rightarrow \\ a_i &= N^1 a_i^1 + N^2 a_i^2 + \dots + N^{nep} a_i^{nep} = \tilde{N}^T \tilde{a}_i^e \rightarrow \quad \tilde{a} = \underline{\tilde{N}} \tilde{a}^e\end{aligned}$$

# Gradient

$$\begin{aligned}\vec{a} &= N^1 \vec{a}^1 + N^2 \vec{a}^2 + \dots + N^{nep} \vec{a}^{nep} = \tilde{N}^T \tilde{a}^e \rightarrow \\ a_i &= N^1 a_i^1 + N^2 a_i^2 + \dots + N^{nep} a_i^{nep} = \tilde{N}^T \tilde{a}_i^e \rightarrow \quad \tilde{a} = \underline{\tilde{N}} \tilde{a}^e\end{aligned}$$

$$\mathbf{L}^c = \vec{\nabla} \vec{a} \rightarrow \underline{\tilde{L}}_t = \underline{\tilde{B}} \tilde{a}^e$$

# Integration

$$\vec{a} = N^1 \vec{a}^1 + N^2 \vec{a}^2 + \dots + N^{nep} \vec{a}^{nep} = \tilde{N}^T \vec{\tilde{a}}^e \rightarrow$$

$$a_i = N^1 a_i^1 + N^2 a_i^2 + \dots + N^{nep} a_i^{nep} = \tilde{N}^T \tilde{a}_i^e \rightarrow \tilde{a} = \underline{N} \tilde{a}^e$$

$$\mathbf{L}^c = \vec{\nabla} \vec{a} \rightarrow \underline{\underline{L}}_t = \underline{B} \tilde{a}^e$$

$$\tilde{w}^{eT} \left[ \int_{V^e} \underline{B}^T \underline{\underline{W}} \underline{B} dV^e \right] \delta \tilde{u}^e = \tilde{w}^{eT} \tilde{f}_e^e - \tilde{w}^{eT} \tilde{f}_i^e$$

$$\tilde{w}^{eT} \left[ \int_{\xi_1=-1}^{\xi_1=1} \int_{\xi_2=-1}^{\xi_2=1} \int_{\xi_3=-1}^{\xi_3=1} \underline{B}^T [\underline{\underline{W}}] \underline{B} \det(\underline{J}) d\xi_1 d\xi_2 d\xi_3 \right] \delta \tilde{u}^e =$$

$$\tilde{w}^{eT} \tilde{f}_e^e - \tilde{w}^{eT} \tilde{f}_i^e$$

$$\tilde{w}^{eT} \underline{K}^e \delta \tilde{u}^e = \tilde{w}^{eT} \tilde{f}_e^e - \tilde{w}^{eT} \tilde{f}_i^e$$

# Integration

$$\begin{aligned}\int_{V^e} g(x_1, x_2, x_3) dV^e &= \int_{\xi_1=-1}^1 \int_{\xi_2=-1}^1 \int_{\xi_3=-1}^1 f(\xi_1, \xi_2, \xi_3) d\xi_1 d\xi_2 d\xi_3 \\ &= \sum_{ip=1}^{nip} c^{ip} f(\xi_1^{ip}, \xi_2^{ip}, \xi_3^{ip})\end{aligned}$$

# Assemblation

$$\sum_e \tilde{w}^{eT} \underline{K}^e \delta \underline{u}^e = \sum_e \tilde{w}^{eT} \underline{f}_e^e - \sum_e \tilde{w}^{eT} \underline{f}_i^e \rightarrow$$

$$\tilde{w}^T \underline{K} \delta \underline{u} = \tilde{w}^T \underline{f}_e - \tilde{w}^T \underline{f}_i = \tilde{w}^T \underline{r} \quad \forall \tilde{w} \rightarrow$$

$$\underline{K} \delta \underline{u} = \underline{f}_e - \underline{f}_i = \underline{r}$$

# Solution

$$\begin{bmatrix} k_{11} & k_{12} & k_{13} & \cdots \\ k_{21} & k_{22} & k_{23} & \cdots \\ k_{31} & k_{32} & k_{33} & \cdots \\ \vdots & \vdots & \vdots & . \end{bmatrix} \begin{bmatrix} a \\ a \\ a \\ \vdots \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \\ 0 \\ \vdots \end{bmatrix} \rightarrow \underline{K} = \text{singular} \rightarrow \det \underline{K} = 0$$

prevent rigid body movement with BC's

other BC's : prescribed displacements / loads / temperature

$$\delta \underline{u} = \underline{K}^{-1} \underline{r}$$

# Program structure

```
read input data from input file
calculate additional variables from input data
initialize values and arrays

while load increments to be done

    for all elements
        for all integration points
            calculate contribution to initial element stiffness matrix
        end integration point loop
        assemble global stiffness matrix
    end element loop

    determine external incremental load from input

    while non-converged iteration step

        take tyings into account
        take boundary conditions into account

        calculate iterative nodal displacements
        calculate total deformation

        for all elements
            for all integration points
                calculate stresses from material behavior
                calculate material stiffness from material behavior
                calculate contribution to element internal nodal forces
                calculate contribution to element stiffness matrix
            end integration point loop
            assemble global stiffness matrix
            assemble global internal load column
        end element loop

        calculate residual load column
        calculate convergence norm

    end iteration step

    store data for post-processing
end load increment
```

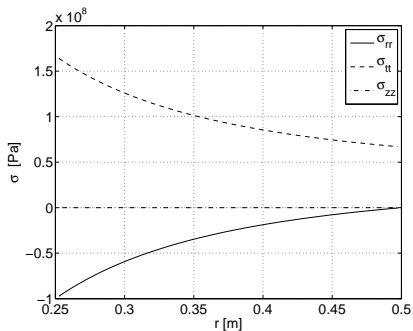
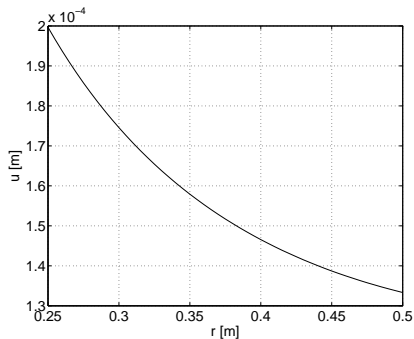


# NUMERICAL SOLUTIONS

[back to index](#)

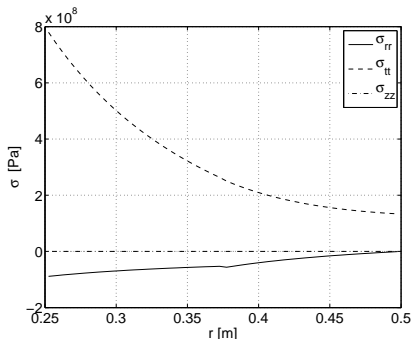
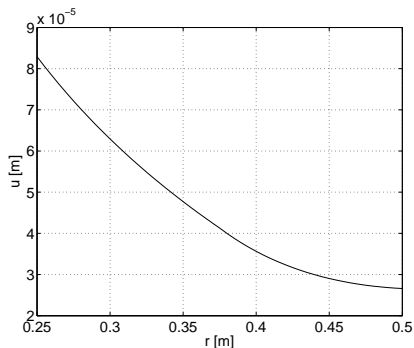
# Example

| isotropic | plane stress |  $p_i = 100 \text{ MPa}$  |  $p_e = 0 \text{ MPa}$  |  
|  $a = 0.25 \text{ m}$  |  $b = 0.5 \text{ m}$  |  $h = 0.5 \text{ m}$  |  $E = 250 \text{ GPa}$  |  $\nu = 0.33$  |



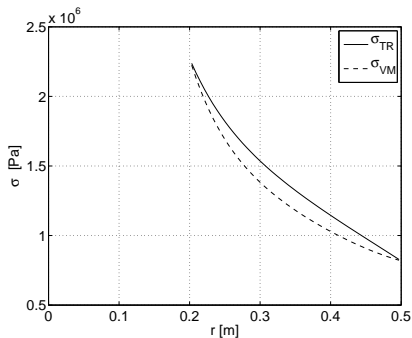
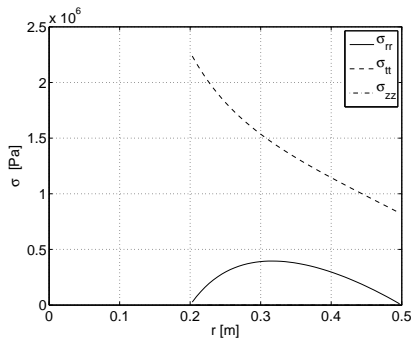
# Compound thick-walled pressurized cylinder

isotropic	plane stress	$p_i = 100 \text{ MPa}$	$p_e = 0 \text{ MPa}$	
$a_1 = 0.25 \text{ m}$	$a_2 = 0.375 \text{ m}$	$E = 250 \text{ GPa}$	$\nu = 0.33$	
$a_2 = 0.375 \text{ m}$	$b = 0.5 \text{ m}$	$E1 = E \text{ GPa}$	$E2 = 10E \text{ GPa}$	
$\nu_{12} = \nu/10$	$\nu_{32} = \nu/10$			



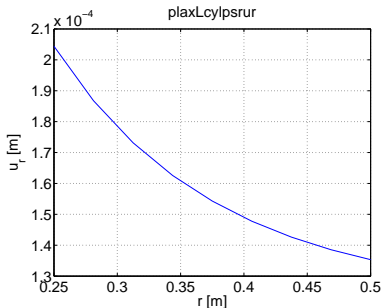
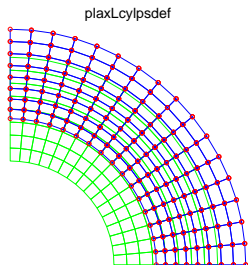
# Rotating disc

| orthotropic | plane stress |  $\omega = 6 \text{ c/s}$  |  
|  $a = 0.2 \text{ m}$  |  $b = 0.5 \text{ m}$  |  $E = 200 \text{ GPa}$  |  $\nu = 0.3$  |  $Gr = 7500 \text{ kg/m}^3$  |

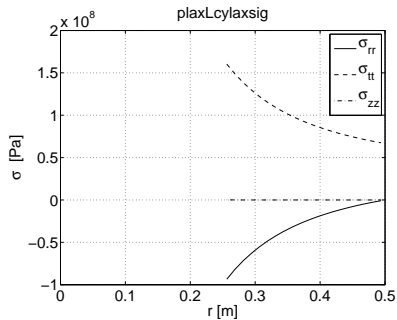
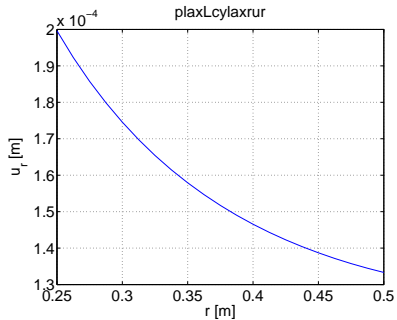


# Thick-walled pressurized cylinder: plane stress

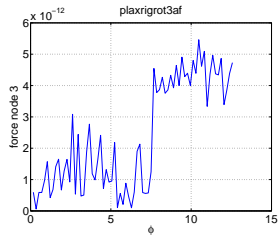
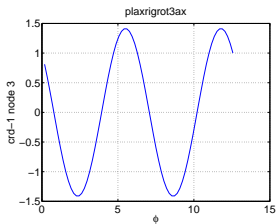
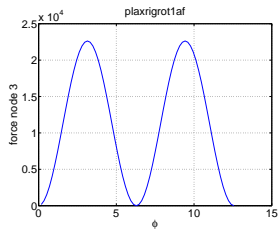
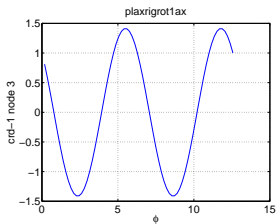
$$\begin{array}{|l|l|l|l|l|l|} \hline a = 0.25 \text{ m} & b = 0.5 \text{ m} & h = 0.5 \text{ m} & E = 250 \text{ GPa} & \nu = 0.33 & \\ \hline p_i = 100 \text{ MPa} & p_e = 0 \text{ MPa} & & & & \\ \hline \end{array}$$



# Thick-walled pressurized cylinder: axi-symmetric



# Rigid rotation



# THREE-DIMENSIONAL MATERIAL MODELS

[back to index](#)



ELASTIC

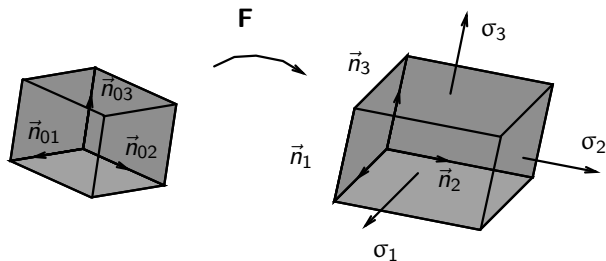
[back to index](#)

# Elastic material behavior

$$\mathbf{P} = \mathbf{G}(\mathbf{E}) \quad \text{with} \quad \mathbf{E} = \frac{1}{2}(\mathbf{C} - \mathbf{I}) = \frac{1}{2}(\mathbf{F}^c \cdot \mathbf{F} - \mathbf{I})$$

$$\begin{aligned} \boldsymbol{\sigma} &= J^{-1} \mathbf{F} \cdot \mathbf{P} \cdot \mathbf{F}^c \\ &= J^{-1} \mathbf{F} \cdot \mathbf{G}(\mathbf{E}) \cdot \mathbf{F}^c \quad \text{with} \quad J = \det(\mathbf{F}) \\ &= \mathbf{K}(\mathbf{A}) \quad \text{with} \quad \mathbf{A} = \frac{1}{2}(\mathbf{B} - \mathbf{I}) = \frac{1}{2}(\mathbf{F} \cdot \mathbf{F}^c - \mathbf{I}) \end{aligned}$$

# Isotropic elastic material models



$$\mathbf{U} = \lambda_1 \vec{n}_{01} \vec{n}_{01} + \lambda_2 \vec{n}_{02} \vec{n}_{02} + \lambda_3 \vec{n}_{03} \vec{n}_{03}$$

$$\mathbf{R} = \vec{n}_1 \vec{n}_{01} + \vec{n}_2 \vec{n}_{02} + \vec{n}_3 \vec{n}_{03}$$

$$\mathbf{F} = \lambda_1 \vec{n}_1 \vec{n}_{01} + \lambda_2 \vec{n}_2 \vec{n}_{02} + \lambda_3 \vec{n}_3 \vec{n}_{03}$$

$$\begin{aligned} \mathbf{P} &= J \mathbf{F}^{-1} \cdot (\sigma_1 \vec{n}_1 \vec{n}_1 + \sigma_2 \vec{n}_2 \vec{n}_2 + \sigma_3 \vec{n}_3 \vec{n}_3) \cdot \mathbf{F}^{-c} \\ &= J \{ \sigma_1 \lambda_1^{-2} \vec{n}_{01} \vec{n}_{01} + \sigma_2 \lambda_2^{-2} \vec{n}_{02} \vec{n}_{02} + \sigma_3 \lambda_3^{-2} \vec{n}_{03} \vec{n}_{03} \} \\ &= s_1 \vec{n}_{01} \vec{n}_{01} + s_2 \vec{n}_{01} \vec{n}_{01} + s_3 \vec{n}_{01} \vec{n}_{01} \end{aligned}$$

## P – E model

$$\begin{aligned}\mathbf{P} &= s_1 \vec{n}_{01} \vec{n}_{01} + s_2 \vec{n}_{02} \vec{n}_{02} + s_3 \vec{n}_{03} \vec{n}_{03} \\ \mathbf{E} &= \varepsilon_1 \vec{n}_{01} \vec{n}_{01} + \varepsilon_2 \vec{n}_{02} \vec{n}_{02} + \varepsilon_3 \vec{n}_{03} \vec{n}_{03}\end{aligned}$$

$$\begin{aligned}\mathbf{P} &= \sum s_i \vec{n}_{0i} \vec{n}_{0i} \\ &= \mathbf{G}(\mathbf{E}) = \sum G(\varepsilon_i) \vec{n}_{0i} \vec{n}_{0i} \\ &= a_0 \mathbf{I} + a_1 \mathbf{E} + a_2 \mathbf{E}^2 + a_3 \mathbf{E}^3 + \dots\end{aligned}$$

Cayley-Hamilton's theorem

$$\mathbf{E}^3 = J_1(\mathbf{E}) \mathbf{E}^2 - J_2(\mathbf{E}) \mathbf{E} + J_3(\mathbf{E}) \mathbf{I}$$

$$\mathbf{P} = \alpha_0 \mathbf{I} + \alpha_1 \mathbf{E} + \alpha_2 \mathbf{E}^2 \quad \text{with} \quad \alpha_i = \alpha_i \{J_1(\mathbf{E}), J_2(\mathbf{E}), J_3(\mathbf{E})\}$$

## $\sigma - \mathbf{A}$ model

$$\sigma = \sigma_1 \vec{n}_1 \vec{n}_1 + \sigma_2 \vec{n}_2 \vec{n}_2 + \sigma_3 \vec{n}_3 \vec{n}_3$$

$$\mathbf{A} = A_1 \vec{n}_1 \vec{n}_1 + A_2 \vec{n}_2 \vec{n}_2 + A_3 \vec{n}_3 \vec{n}_3$$

$$\begin{aligned}\sigma &= \sum \sigma_i \vec{n}_i \vec{n}_i \\ &= \mathbf{K}(\mathbf{A}) = \sum K(A_i) \vec{n}_i \vec{n}_i \\ &= b_0 \mathbf{I} + b_1 \mathbf{A} + b_2 \mathbf{A}^2 + b_3 \mathbf{A}^3 + \dots\end{aligned}$$

Cayley-Hamilton's theorem

$$\mathbf{A}^3 = J_1(\mathbf{A}) \mathbf{A}^2 - J_2(\mathbf{A}) \mathbf{A} + J_3(\mathbf{A}) \mathbf{I}$$

$$\sigma = \beta_0 \mathbf{I} + \beta_1 \mathbf{A} + \beta_2 \mathbf{A}^2 \quad \text{with} \quad \beta_i = \beta_i \{J_1(\mathbf{A}), J_2(\mathbf{A}), J_3(\mathbf{A})\}$$

# Isotropic elastic material : $\sigma - \mathbf{A}$

$$\begin{aligned}\sigma &= J^{-1} \mathbf{F} \cdot \mathbf{P} \cdot \mathbf{F}^c \\&= J^{-1} \mathbf{F} \cdot [\alpha_0 \mathbf{I} + \alpha_1 \mathbf{E} + \alpha_2 \mathbf{E}^2] \cdot \mathbf{F}^c \\&= J^{-1} \mathbf{F} \cdot [(\alpha_0 - \tfrac{1}{2}\alpha_1 + \alpha_2) \mathbf{I} + (\tfrac{1}{2}\alpha_1 - \tfrac{1}{2}\alpha_2) \mathbf{C} + \tfrac{1}{4}\alpha_2 \mathbf{C}^2] \cdot \mathbf{F}^c \\&= \{J_3(\mathbf{B})\}^{-1/2} [(\alpha_0 - \tfrac{1}{2}\alpha_1 + \alpha_2) \mathbf{B} + (\tfrac{1}{2}\alpha_1 - \tfrac{1}{2}\alpha_2) \mathbf{B}^2 + \tfrac{1}{4}\alpha_2 \mathbf{B}^3] \\&\quad \mathbf{B}^3 = J_1(\mathbf{B}) \mathbf{B}^2 - J_2(\mathbf{B}) \mathbf{B} + J_3(\mathbf{B}) \mathbf{I} \\&= J_3^{-1/2} [(\tfrac{1}{2}\alpha_1 - \tfrac{1}{2}\alpha_2 + \tfrac{1}{4}\alpha_2 J_1) \mathbf{B}^2 + \\&\quad (\alpha_0 - \tfrac{1}{2}\alpha_1 + \alpha_2 - \tfrac{1}{4}\alpha_2 J_2) \mathbf{B} + \tfrac{1}{4}\alpha_2 J_3 \mathbf{I}] \\&\quad \mathbf{A} = \tfrac{1}{2}(\mathbf{B} - \mathbf{I}) \quad \rightarrow \quad \mathbf{B} = 2\mathbf{A} + \mathbf{I} \\&\quad \mathbf{A}^2 = \tfrac{1}{4}\mathbf{B}^2 - \tfrac{1}{2}\mathbf{B} + \tfrac{1}{4}\mathbf{I} \quad \rightarrow \quad \mathbf{B}^2 = 4\mathbf{A}^2 + 2\mathbf{B} - \mathbf{I} \\&= J_3^{-1/2} [(2\alpha_1 - 2\alpha_2 + \alpha_2 J_1) \mathbf{A}^2 + (\alpha_0 + \tfrac{1}{2}\alpha_1 + \tfrac{1}{2}\alpha_2 J_1 - \tfrac{1}{4}\alpha_2 J_2) \mathbf{A} + \\&\quad (\alpha_0 + \alpha_1 - \tfrac{1}{2}\alpha_2 + \tfrac{3}{4}\alpha_2 J_1 - \tfrac{1}{4}\alpha_2 J_2 + \tfrac{1}{4}\alpha_2 J_3) \mathbf{I}] \\&= \beta_2 \mathbf{A}^2 + \beta_1 \mathbf{A} + \beta_0 \mathbf{I}\end{aligned}$$

## Linear $\mathbf{P} - \mathbf{E}$ model

$$\mathbf{P} = \alpha_0 \mathbf{I} + \alpha_1 \mathbf{E} + \alpha_2 \mathbf{E}^2 \quad \text{with} \quad \alpha_i = \alpha_i \{J_1(\mathbf{E}), J_2(\mathbf{E}), J_3(\mathbf{E})\}$$

linear  $\rightarrow$

1.  $\alpha_2 = 0$
2.  $\alpha_1 = \text{constant} = c_1$
3.  $\alpha_0 = \text{linear in } \mathbf{E} = c_0 \text{tr}(\mathbf{E})$

$$\mathbf{P} = c_0 \text{tr}(\mathbf{E}) \mathbf{I} + c_1 \mathbf{E}$$

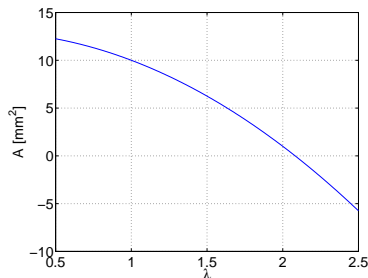
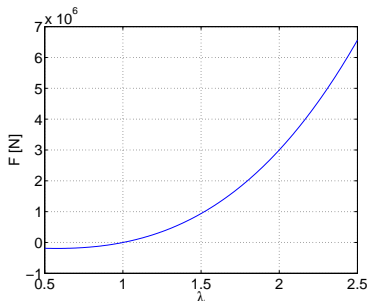
# Tensile test

$$\left. \begin{aligned} P &= c_0 \frac{1}{2}(\lambda^2 - 1) + 2c_0 \frac{1}{2}(\mu^2 - 1) + c_1 \frac{1}{2}(\lambda^2 - 1) \\ 0 &= c_0 \frac{1}{2}(\lambda^2 - 1) + 2c_0 \frac{1}{2}(\mu^2 - 1) + c_1 \frac{1}{2}(\mu^2 - 1) \end{aligned} \right\} \rightarrow$$

$$\frac{1}{2}(\mu^2 - 1) = -\frac{c_0}{2c_0 + c_1} \frac{1}{2}(\lambda^2 - 1) = -\nu \frac{1}{2}(\lambda^2 - 1)$$

$$P = \frac{c_1(3c_0 + c_1)}{2c_0 + c_1} \frac{1}{2}(\lambda^2 - 1) = E \frac{1}{2}(\lambda^2 - 1)$$

$$F = \sigma A = \frac{\lambda}{\mu^2} P \mu^2 A_0 = \lambda P A_0 = \frac{1}{2} \lambda (\lambda^2 - 1) E A_0$$





# Simple shear test : plane stress

$$\sigma_{33} = P_{33} = 0 \rightarrow c_0(E_{11} + E_{22} + E_{33}) + c_1 E_{33} = 0 \rightarrow E_{33} = -\frac{c_0}{c_0 + c_1} (E_{11} + E_{22})$$

$$\mathbf{F} = \mathbf{I} + (F_{33} - 1)\vec{e}_3\vec{e}_3 + \gamma\vec{e}_1\vec{e}_2$$

$$\mathbf{E} = \frac{1}{2}(\mathbf{F}^c \cdot \mathbf{F} - \mathbf{I}) = \frac{1}{2} [\gamma^2 \vec{e}_2 \vec{e}_2 + \gamma(\vec{e}_1 \vec{e}_2 + \vec{e}_2 \vec{e}_1) + \{2(F_{33} - 1) + (F_{33} - 1)^2\} \vec{e}_3 \vec{e}_3]$$

$$F_{33} = \sqrt{2E_{33} + 1} \rightarrow J = \det(\mathbf{F}) = F_{33} = \sqrt{2E_{33} + 1}$$

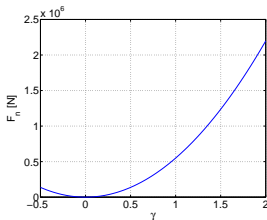
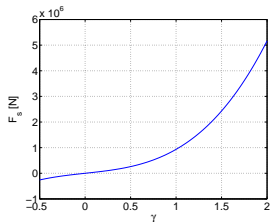
$$\mathbf{P} = \frac{c_0 c_1}{c_0 + c_1} (E_{11} + E_{22}) + c_1 \mathbf{E}$$

$$= \frac{c_0 c_1}{c_0 + c_1} \frac{1}{2} \gamma^2 \mathbf{I} + c_1 \frac{1}{2} \gamma^2 \vec{e}_2 \vec{e}_2 + c_1 \frac{1}{2} \gamma (\vec{e}_1 \vec{e}_2 + \vec{e}_2 \vec{e}_1)$$

$$\boldsymbol{\sigma} = J^{-1} \mathbf{F} \cdot \mathbf{P} \cdot \mathbf{F}^c = J^{-1} [\mathbf{P} + (\gamma P_{12} + \gamma P_{21} + \gamma^2 P_{22}) \vec{e}_1 \vec{e}_1 + \gamma P_{22} (\vec{e}_1 \vec{e}_2 + \vec{e}_2 \vec{e}_1)]$$

$$p_n = \vec{e}_2 \cdot \boldsymbol{\sigma} \cdot \vec{e}_2 = \sigma_{22} \quad ; \quad p_s = \vec{e}_1 \cdot \boldsymbol{\sigma} \cdot \vec{e}_2 = \sigma_{12}$$

$$F_n = p_n dw_0 = p_n F_{33} d_0 w_0 \quad ; \quad F_s = p_s dw_0 = p_s F_{33} d_0 w_0$$



## Linear $\sigma - \mathbf{A}$ model

$$\sigma = \beta_0 \mathbf{I} + \beta_1 \mathbf{A} + \beta_2 \mathbf{A}^2 \quad \text{with} \quad \beta_i = \beta_i \{J_1(\mathbf{A}), J_2(\mathbf{A}), J_3(\mathbf{A})\}$$

linear  $\rightarrow$

1.  $\beta_2 = 0$
2.  $\beta_1 = \text{constant} = c_1$
3.  $\beta_0 = \text{linear in } \mathbf{A} = c_0 \text{tr}(\mathbf{A})$

$$\sigma = c_0 \text{tr}(\mathbf{A}) \mathbf{I} + c_1 \mathbf{A}$$

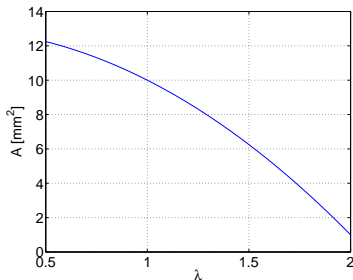
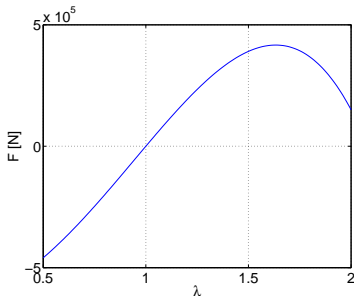
# Tensile test

$$\left. \begin{aligned} \sigma &= c_0 \frac{1}{2}(\lambda^2 - 1) + 2c_0 \frac{1}{2}(\mu^2 - 1) + c_1 \frac{1}{2}(\lambda^2 - 1) \\ 0 &= c_0 \frac{1}{2}(\lambda^2 - 1) + 2c_0 \frac{1}{2}(\mu^2 - 1) + c_1 \frac{1}{2}(\mu^2 - 1) \end{aligned} \right\} \rightarrow$$

$$\frac{1}{2}(\mu^2 - 1) = -\frac{c_0}{2c_0 + c_1} \frac{1}{2}(\lambda^2 - 1) = -\nu \frac{1}{2}(\lambda^2 - 1)$$

$$\sigma = \frac{c_0(3c_0 + c_1)}{2c_0 + c_1} \frac{1}{2}(\lambda^2 - 1) = E \frac{1}{2}(\lambda^2 - 1)$$

$$F = \sigma A = \sigma \mu^2 A_0 = \frac{1}{2}(\lambda^2 - 1)\{1 - \nu(\lambda^2 - 1)\} EA_0$$



# Simple shear test : plane strain

$$\mathbf{F} = \mathbf{I} + \gamma \vec{e}_1 \vec{e}_2$$

$$\mathbf{B} = \mathbf{F} \cdot \mathbf{F}^c = \mathbf{I} + \gamma^2 \vec{e}_1 \vec{e}_1 + \gamma (\vec{e}_1 \vec{e}_2 + \vec{e}_2 \vec{e}_1)$$

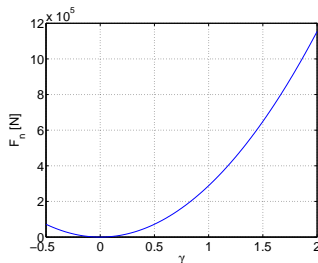
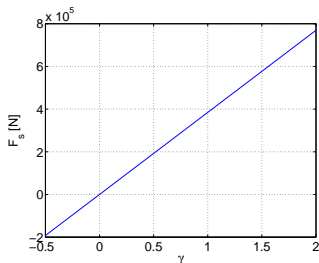
$$\mathbf{A} = \frac{1}{2}(\mathbf{B} - \mathbf{I}) = \frac{1}{2} \gamma^2 \vec{e}_1 \vec{e}_1 + \frac{1}{2} \gamma (\vec{e}_1 \vec{e}_2 + \vec{e}_2 \vec{e}_1)$$

$$\boldsymbol{\sigma} = c_0 \frac{1}{2} \gamma^2 \mathbf{I} + c_1 \frac{1}{2} \gamma^2 \vec{e}_1 \vec{e}_1 + c_1 \frac{1}{2} \gamma (\vec{e}_1 \vec{e}_2 + \vec{e}_2 \vec{e}_1)$$

$$\sigma_{33} = c_0 \frac{1}{2} \gamma^2$$

$$p_n = \vec{e}_2 \cdot \boldsymbol{\sigma} \cdot \vec{e}_2 = c_0 \frac{1}{2} \gamma^2 \quad ; \quad p_s = \vec{e}_1 \cdot \boldsymbol{\sigma} \cdot \vec{e}_2 = c_1 \frac{1}{2} \gamma$$

$$F_n = p_n d_0 w_0 \quad ; \quad F_s = p_s d_0 w_0$$



## Simple shear test : plane stress

$$\sigma_{33} = c_0(A_{11} + A_{22} + A_{33}) + c_1 A_{33} = 0 \rightarrow$$

$$A_{33} = -\frac{c_0}{c_0 + c_1} (A_{11} + A_{22}) \rightarrow F_{33} = \sqrt{2A_{33} + 1}$$

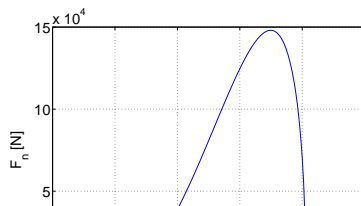
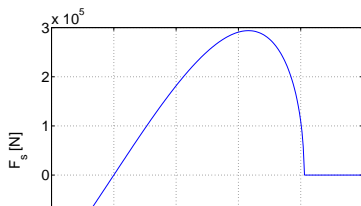
$$\boldsymbol{\sigma} = \frac{c_0 c_1}{c_0 + c_1} (A_{11} + A_{22}) \mathbf{I} + c_1 \mathbf{A}$$

$$\mathbf{A} = \frac{1}{2} \gamma^2 \vec{e}_1 \vec{e}_1 + \frac{1}{2} \gamma (\vec{e}_1 \vec{e}_2 + \vec{e}_2 \vec{e}_1)$$

$$\boldsymbol{\sigma} = \frac{c_0 c_1}{c_0 + c_1} \frac{1}{2} \gamma^2 \mathbf{I} + c_1 \frac{1}{2} \gamma^2 \vec{e}_1 \vec{e}_1 + c_1 \frac{1}{2} \gamma (\vec{e}_1 \vec{e}_2 + \vec{e}_2 \vec{e}_1)$$

$$p_n = \vec{e}_2 \cdot \boldsymbol{\sigma} \cdot \vec{e}_2 = \frac{c_0 c_1}{c_0 + c_1} \frac{1}{2} \gamma^2 \quad ; \quad p_s = \vec{e}_1 \cdot \boldsymbol{\sigma} \cdot \vec{e}_2 = c_1 \frac{1}{2} \gamma$$

$$F_n = p_n dw_0 = p_n F_{33} d_0 w_0 \quad ; \quad F_s = p_s dw_0 = p_s F_{33} d_0 w_0$$



# Hyper-elastic material models

$$\phi = \phi(\mathbf{E}) \quad \rightarrow \quad W = W(\mathbf{C}) \quad \rightarrow$$

$$\mathbf{P} = \frac{d\phi(d\mathbf{E})}{d\mathbf{E}} = \frac{dW(\mathbf{C})}{d\mathbf{C}} : \frac{d\mathbf{C}}{d\mathbf{E}} = 2 \frac{dW(\mathbf{C})}{d\mathbf{C}} = \mathbf{G}(\mathbf{E})$$

$$\sigma = \frac{1}{J} \mathbf{F} \cdot \mathbf{P} \cdot \mathbf{F}^c = \frac{2}{J} \mathbf{F} \cdot \frac{dW(\mathbf{C})}{d\mathbf{C}} \cdot \mathbf{F}^c$$

## Isotropic hyper-elastic model : $\mathbf{P} - \mathbf{E}$

$$\phi = \phi(\mathbf{E}) = \phi\{J_1(\mathbf{E}), J_2(\mathbf{E}), J_3(\mathbf{E})\} \rightarrow$$

$$\mathbf{P} = \frac{\partial \phi}{\partial J_1} \frac{dJ_1}{d\mathbf{E}} + \frac{\partial \phi}{\partial J_2} \frac{dJ_2}{d\mathbf{E}} + \frac{\partial \phi}{\partial J_3} \frac{dJ_3}{d\mathbf{E}}$$

$$\frac{dJ_1}{d\mathbf{E}} = \mathbf{I} \quad ; \quad \frac{dJ_2}{d\mathbf{E}} = J_1 \mathbf{I} - \mathbf{E} \quad ; \quad \frac{dJ_3}{d\mathbf{E}} = J_2 \mathbf{I} - J_1 \mathbf{E} + \mathbf{E}^2 \quad \rightarrow$$

$$\begin{aligned} \mathbf{P} &= \left( \frac{\partial \phi}{\partial J_1} + \frac{\partial \phi}{\partial J_2} J_1 + \frac{\partial \phi}{\partial J_3} J_2 \right) \mathbf{I} + \left( -\frac{\partial \phi}{\partial J_2} - \frac{\partial \phi}{\partial J_3} J_1 \right) \mathbf{E} + \frac{\partial \phi}{\partial J_3} \mathbf{E}^2 \\ &= \alpha_0 \mathbf{I} + \alpha_1 \mathbf{E} + \alpha_2 \mathbf{E}^2 \end{aligned}$$

## Isotropic hyper-elastic model : $\mathbf{P} - \mathbf{C}$

$$W = W(\mathbf{C}) = W\{J_1(\mathbf{C}), J_2(\mathbf{C}), J_3(\mathbf{C})\} \rightarrow$$

$$\mathbf{P} = 2 \left( \frac{\partial W}{\partial J_1} \frac{dJ_1}{d\mathbf{C}} + \frac{\partial W}{\partial J_2} \frac{dJ_2}{d\mathbf{C}} + \frac{\partial W}{\partial J_3} \frac{dJ_3}{d\mathbf{C}} \right)$$

$$\frac{dJ_1}{d\mathbf{C}} = \mathbf{I} \quad ; \quad \frac{dJ_2}{d\mathbf{C}} = J_1 \mathbf{I} - \mathbf{C} \quad ; \quad \frac{dJ_3}{d\mathbf{C}} = J_2 \mathbf{I} - J_1 \mathbf{C} + \mathbf{C}^2 \quad \rightarrow$$

$$\begin{aligned} \mathbf{P} &= 2 \left( \frac{\partial W}{\partial J_1} + \frac{\partial W}{\partial J_2} J_1 + \frac{\partial W}{\partial J_3} J_2 \right) \mathbf{I} + 2 \left( -\frac{\partial W}{\partial J_2} - \frac{\partial W}{\partial J_3} J_1 \right) \mathbf{C} + 2 \frac{\partial W}{\partial J_3} \mathbf{C}^2 \\ &= \bar{\alpha}_0 \mathbf{I} + \bar{\alpha}_1 \mathbf{E} + \bar{\alpha}_2 \mathbf{E}^2 \end{aligned}$$



## Isotropic hyper-elastic model : $\sigma - \mathbf{A}$ model

$$\begin{aligned}
 \boldsymbol{\sigma} &= \frac{1}{J} \mathbf{F} \cdot \mathbf{P} \cdot \mathbf{F}^c = \frac{2}{J} \mathbf{F} \cdot \frac{dW(\mathbf{C})}{d\mathbf{C}} \cdot \mathbf{F}^c \\
 &= \frac{2}{\sqrt{J_3}} \mathbf{F} \cdot \left( \frac{\partial W}{\partial J_1} \frac{dJ_1}{d\mathbf{C}} + \frac{\partial W}{\partial J_2} \frac{dJ_2}{d\mathbf{C}} + \frac{\partial W}{\partial J_3} \frac{dJ_3}{d\mathbf{C}} \right) \cdot \mathbf{F}^c \\
 &= \frac{2}{\sqrt{J_3}} \mathbf{F} \cdot \left\{ \left( \frac{\partial W}{\partial J_1} + J_1 \frac{\partial W}{\partial J_2} + J_2 \frac{\partial W}{\partial J_3} \right) \mathbf{I} + \left( -\frac{\partial W}{\partial J_2} - J_1 \frac{\partial W}{\partial J_3} \right) \mathbf{C} + \left( \frac{\partial W}{\partial J_3} \right) \mathbf{C}^2 \right\} \\
 &= \frac{2}{\sqrt{J_3}} \mathbf{F} \cdot (\gamma_0 \mathbf{I} + \gamma_1 \mathbf{C} + \gamma_2 \mathbf{C}^2) \cdot \mathbf{F}^c = \frac{2}{\sqrt{J_3}} (\gamma_0 \mathbf{B} + \gamma_1 \mathbf{B}^2 + \gamma_2 \mathbf{B}^3) \\
 &\quad \mathbf{B}^3 = J_1 \mathbf{B}^2 - J_2 \mathbf{B} + J_3 \mathbf{I} \\
 &= \frac{2}{\sqrt{J_3}} [(\gamma_1 + \gamma_2 J_1) \mathbf{B}^2 + (\gamma_0 - \gamma_2 J_2) \mathbf{B} + (\gamma_2 J_3) \mathbf{I}] \\
 &\quad \mathbf{A} = \frac{1}{2}(\mathbf{B} - \mathbf{I}) \quad \rightarrow \quad \mathbf{B} = 2\mathbf{A} + \mathbf{I} \quad \rightarrow \quad \mathbf{B}^2 = 4\mathbf{A}^2 + 2\mathbf{B} - \mathbf{I} \\
 &= \frac{2}{\sqrt{J_3}} [(4\gamma_1 + 4\gamma_2 J_1) \mathbf{A}^2 + (\gamma_0 + 2\gamma_1 + 2\gamma_2 J_1 - \gamma_2 J_2) \mathbf{A} + \\
 &\quad (\gamma_0 - \gamma_1 + \gamma_2 J_1 - \gamma_2 J_2 + \gamma_2 J_3) \mathbf{I}] \\
 &= \beta_2 \mathbf{A}^2 + \beta_1 \mathbf{A} + \beta_0 \mathbf{I}
 \end{aligned}$$

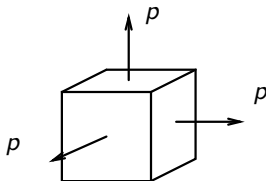
# Incompressibility

$$J = \det(\mathbf{F}) = 1 \quad \rightarrow \quad \det(\mathbf{C}) = J_3(\mathbf{C}) = 1 \quad \rightarrow \quad W(\mathbf{C}) = W\{J_1(\mathbf{C}), J_2(\mathbf{C})\}$$

$$\mathbf{P} = 2 \left( \frac{\partial W}{\partial J_1} \frac{dJ_1}{d\mathbf{C}} + \frac{\partial W}{\partial J_2} \frac{dJ_2}{d\mathbf{C}} \right) = 2 \left\{ \left( \frac{\partial W}{\partial J_1} + \frac{\partial W}{\partial J_2} J_1 \right) \mathbf{I} - \frac{\partial W}{\partial J_2} \mathbf{C} \right\}$$

$$\boldsymbol{\sigma} = \mathbf{F} \cdot \mathbf{P} \cdot \mathbf{F}^c = 2 \left\{ \left( \frac{\partial W}{\partial J_1} + \frac{\partial W}{\partial J_2} J_1 \right) \mathbf{B} - \frac{\partial W}{\partial J_2} \mathbf{B}^2 \right\}$$

# Incompressibility



$$\begin{aligned}\boldsymbol{\sigma} &= -p\mathbf{I} + \mathbf{F} \cdot \mathbf{P} \cdot \mathbf{F}^c \\ &= -p\mathbf{I} + 2 \left\{ \left( \frac{\partial W}{\partial J_1} + \frac{\partial W}{\partial J_2} J_1 \right) \mathbf{B} - \frac{\partial W}{\partial J_2} \mathbf{B}^2 \right\} \\ &= -p\mathbf{I} + \boldsymbol{\tau}\end{aligned}$$

hydrostatic pressure	:	$p$	:	extra unknown
incompressibility condition	:	$\det(\mathbf{F}) = J = 1$	:	extra equation

# Rivlin models

$$W(\mathbf{C}) = \sum_{i=0}^m \sum_{j=0}^n c_{ij} \{J_1(\mathbf{C}) - 3\}^i \{J_2(\mathbf{C}) - 3\}^j \quad \text{with} \quad C_{00} = 0$$

$$J_1 = \text{tr}(\mathbf{C}) = \lambda_1^2 + \lambda_2^2 + \lambda_3^2$$

$$J_2 = \frac{1}{2} \{ \text{tr}^2(\mathbf{C}) - \text{tr}(\mathbf{C}^2) \}$$

$$= \frac{1}{2} \{ (\lambda_1^2 + \lambda_2^2 + \lambda_3^2)^2 - (\lambda_1^4 + \lambda_2^4 + \lambda_3^4) \}$$

$$= \lambda_1^2 \lambda_2^2 + \lambda_2^2 \lambda_3^2 + \lambda_3^2 \lambda_1^2$$

$$J_3 = \det(\mathbf{C}) = \lambda_1^2 \lambda_2^2 \lambda_3^2 = 1$$

$$W(\mathbf{C}) = \sum_{i=0}^m \sum_{j=0}^n c_{ij} (\lambda_1^2 + \lambda_2^2 + \lambda_3^2 - 3)^i \left( \frac{1}{\lambda_1^2} + \frac{1}{\lambda_2^2} + \frac{1}{\lambda_3^2} - 3 \right)^j$$

# Neo-Hookean model

$$W = C_{10}(J_1 - 3)$$

$$\boldsymbol{\sigma} = -p\mathbf{I} + 2C_{10}\mathbf{B}$$

## Tensile test

$$\mathbf{B} = \lambda^2 \vec{e}_1 \vec{e}_1 + \mu^2 (\vec{e}_2 \vec{e}_2 + \vec{e}_3 \vec{e}_3) = \lambda^2 \vec{e}_1 \vec{e}_1 + \frac{1}{\lambda} (\vec{e}_2 \vec{e}_2 + \vec{e}_3 \vec{e}_3)$$

$$\sigma = -p \mathbf{I} + 2C_{10} \lambda^2 \vec{e}_1 \vec{e}_1 + 2C_{10} \frac{1}{\lambda} (\vec{e}_2 \vec{e}_2 + \vec{e}_3 \vec{e}_3)$$

$$\left. \begin{array}{l} \sigma = -p + 2C_{10} \lambda^2 \\ 0 = -p + 2C_{10} \frac{1}{\lambda} \end{array} \right\} \rightarrow$$

$$\sigma = 2C_{10} \left( \lambda^2 - \frac{1}{\lambda} \right)$$

$$F = \sigma A = \sigma \mu^2 A_0 = \sigma \frac{1}{\lambda} A_0 = 2C_{10} A_0 \left( \lambda - \frac{1}{\lambda^2} \right)$$

# Mooney-Rivlin material model

$$W = C_{10}(J_1 - 3) + C_{01}(J_2 - 3)$$

$$\boldsymbol{\sigma} = -p\mathbf{I} + 2\{C_{10} + C_{01}\text{tr}(\mathbf{B})\}\mathbf{B} - 2C_{01}\mathbf{B}^2$$

## Tensile test

$$\mathbf{B} = \lambda^2 \vec{e}_1 \vec{e}_1 + \mu^2 (\vec{e}_2 \vec{e}_2 + \vec{e}_3 \vec{e}_3) = \lambda^2 \vec{e}_1 \vec{e}_1 + \frac{1}{\lambda} (\vec{e}_2 \vec{e}_2 + \vec{e}_3 \vec{e}_3)$$

$$\text{tr}(\mathbf{B}) = \lambda^2 + \frac{2}{\lambda}$$

$$\mathbf{B}^2 = \lambda^4 \vec{e}_1 \vec{e}_1 + \frac{1}{\lambda^2} (\vec{e}_2 \vec{e}_2 + \vec{e}_3 \vec{e}_3)$$

$$\begin{aligned} \boldsymbol{\sigma} = & -p \mathbf{I} + 2\{C_{10} + C_{01}(\lambda^2 + \frac{2}{\lambda})\} \{\lambda^2 \vec{e}_1 \vec{e}_1 + \frac{1}{\lambda} (\vec{e}_2 \vec{e}_2 + \vec{e}_3 \vec{e}_3)\} \\ & - 2C_{01} \{\lambda^4 \vec{e}_1 \vec{e}_1 + \frac{1}{\lambda^2} (\vec{e}_2 \vec{e}_2 + \vec{e}_3 \vec{e}_3)\} \end{aligned}$$

$$\left. \begin{aligned} \sigma = & -p + 2\{C_{10} + C_{01}(\lambda^2 + \frac{2}{\lambda})\} \lambda^2 - 2C_{01} \lambda^4 \\ 0 = & -p + 2\{C_{10} + C_{01}(\lambda^2 + \frac{2}{\lambda})\} \frac{1}{\lambda} - 2C_{01} \frac{1}{\lambda^2} \end{aligned} \right\} \rightarrow$$

$$\sigma = 2C_{10}(\lambda^2 - \frac{1}{\lambda}) + 2C_{01}(\lambda - \frac{1}{\lambda^2})$$

$$F = \sigma A = \sigma \mu^2 A_0 = \sigma \frac{1}{\lambda} A_0 = 2A_0 \{C_{10}(\lambda - \frac{1}{\lambda^2}) + C_{01}(1 - \frac{1}{\lambda^3})\}$$



## Other energy functions

3-term Mooney-Rivlin  $W = c_{10}(J_1 - 3) + c_{01}(J_2 - 3) + c_{11}(J_1 - 3)(J_2 - 3)$

Signiorini  $W = c_{10}(J_1 - 3) + c_{01}(J_2 - 3) + c_{20}(J_1 - 3)^2$

Yeoh  $W = c_{10}(J_1 - 3) + c_{20}(J_1 - 3)^2 + c_{30}(J_1 - 3)^3$

2nd-order invariant model

$$W = c_{10}(J_1 - 3) + c_{01}(J_2 - 3) + c_{11}(J_1 - 3)(J_2 - 3) + c_{20}(J_1 - 3)^2$$

Kloaner-Segal

$$W = c_{10}(J_1 - 3) + c_{01}(J_2 - 3) + c_{20}(J_1 - 3)^2 + c_{03}(J_2 - 3)^3$$

James, Green, Simpson (3rd-order deformation model)

$$W = c_{10}(J_1 - 3) + c_{01}(J_2 - 3) + c_{11}(J_1 - 3)(J_2 - 3) + c_{20}(J_1 - 3)^2 + c_{30}(J_1 - 3)^3$$

# Ogden models

$$W = \sum_{n=1}^N \frac{\mu_n}{\alpha_n} J^{\frac{-\alpha_n}{3}} (\lambda_1^{\alpha_n} + \lambda_2^{\alpha_n} + \lambda_3^{\alpha_n} - 3) + 4.5K \left(1 - J^{\frac{1}{3}}\right)^2$$

$\mu_n$  : moduli  
 $\alpha_n$  : exponents  
 $K$  : bulk modulus  
 $J$  : volume ratio =  $\det(\mathbf{F})$

foam model

$$W = \sum_{n=1}^N \frac{\mu_n}{\alpha_n} (\lambda_1^{\alpha_n} + \lambda_2^{\alpha_n} + \lambda_3^{\alpha_n} - 3) + \sum_{n=1}^N \frac{\mu_n}{\beta_n} (1 - J^{\beta_n})$$

# Linear $\mathbf{P} - \mathbf{E}$ model

- stress update
- consistent material stiffness tensor for  $\delta \mathbf{P}$   $\rightarrow$   
Total Lagrange formulation
- consistent material stiffness tensor for  $\delta \boldsymbol{\sigma}$   $\rightarrow$   
Updated Lagrange formulation

# Stress update

$$\mathbf{P} = c_0 \text{tr}(\mathbf{E}) \mathbf{I} + c_1 \mathbf{E}$$

$$\text{with} \quad \mathbf{E} = \frac{1}{2}(\mathbf{C} - \mathbf{I})$$

$$= \frac{1}{2}c_0 \mathbf{C} : \mathbf{I} \mathbf{I} + \frac{1}{2}c_1 \mathbf{C} - \frac{1}{2}(3c_0 + c_1) \mathbf{I}$$

$$\text{with} \quad \mathbf{C} = \mathbf{F}^c \cdot \mathbf{F}$$

$$\boldsymbol{\sigma} = J^{-1} \mathbf{F} \cdot \mathbf{P} \cdot \mathbf{F}^c = J^{-1} \mathbf{F} \cdot (\mathbf{P} \cdot \mathbf{F}^c) = J^{-1} \mathbf{F} \cdot (\mathbf{F} \cdot \mathbf{P}^c)^c$$

# Stiffness

$$\delta \mathbf{P} = \frac{1}{2} c_0 \delta \mathbf{C} : \mathbb{I} + \frac{1}{2} c_1 \delta \mathbf{C}$$

$$\mathbf{C} = \mathbf{F}^c \cdot \mathbf{F} \quad \rightarrow \quad \delta \mathbf{C} = \delta \mathbf{F}^c \cdot \mathbf{F} + \mathbf{F}^c \cdot \delta \mathbf{F}$$

$$= \frac{1}{2} c_0 (\delta \mathbf{F}^c \cdot \mathbf{F} + \mathbf{F}^c \cdot \delta \mathbf{F}) : \mathbb{I} + \frac{1}{2} c_1 (\delta \mathbf{F}^c \cdot \mathbf{F} + \mathbf{F}^c \cdot \delta \mathbf{F})$$

$$= c_0 (\mathbf{F}^c \cdot \delta \mathbf{F}) : \mathbb{I} + \frac{1}{2} c_1 (\delta \mathbf{F}^c \cdot \mathbf{F} + \mathbf{F}^c \cdot \delta \mathbf{F})$$

$$= c_0 \mathbf{I} (\mathbf{F}^c : \delta \mathbf{F}) + \frac{1}{2} c_1 \{ (\mathbf{F}^c \cdot \delta \mathbf{F})^c + (\mathbf{F}^c \cdot \delta \mathbf{F}) \}$$

$$\boldsymbol{\sigma} = J^{-1} \mathbf{F} \cdot \mathbf{P} \cdot \mathbf{F}^c \quad \rightarrow$$

$$\delta \boldsymbol{\sigma} = J^{-1} [-\delta J \mathbf{F} \cdot \mathbf{P} \cdot \mathbf{F}^c + \delta \mathbf{F} \cdot \mathbf{P} \cdot \mathbf{F}^c + \mathbf{F} \cdot \delta \mathbf{P} \cdot \mathbf{F}^c + \mathbf{F} \cdot \mathbf{P} \cdot \delta \mathbf{F}^c]$$

$$\delta J = J \text{tr}(\mathbf{L}) = J \mathbf{L} : \mathbf{I} \quad ; \quad \delta \mathbf{F} = \mathbf{L} \cdot \mathbf{F}$$

$$= J^{-1} [-(\mathbf{L} : \mathbf{I}) \mathbf{F} \cdot \mathbf{P} \cdot \mathbf{F}^c + (\mathbf{L} \cdot \mathbf{F}) \cdot \mathbf{P} \cdot \mathbf{F}^c +$$

$$\mathbf{F} \cdot \delta \mathbf{P} \cdot \mathbf{F}^c + \mathbf{F} \cdot \mathbf{P} \cdot (\mathbf{F}^c \cdot \mathbf{L}^c)]$$

$$= -(\mathbf{L} : \mathbf{I}) \boldsymbol{\sigma} + \mathbf{L} \cdot \boldsymbol{\sigma} + \boldsymbol{\sigma} \cdot \mathbf{L}^c + J^{-1} \mathbf{F} \cdot \delta \mathbf{P} \cdot \mathbf{F}^c$$

$$= -\boldsymbol{\sigma} (\mathbf{I} : \mathbf{L}) + (\boldsymbol{\sigma}^c \cdot \mathbf{L}^c)^c + \boldsymbol{\sigma} \cdot \mathbf{L}^c + J^{-1} \mathbf{F} \cdot (\mathbf{F} \cdot \delta \mathbf{P}^c)^c$$

## Matrix/column notation

$$\mathbf{P} = \frac{1}{2}c_0\mathbf{C} : \mathbf{I}\mathbf{I} + \frac{1}{2}c_1\mathbf{C} - \frac{1}{2}(3c_0 + c_1)\mathbf{I} \quad \text{with} \quad \mathbf{C} = \mathbf{F}^c \cdot \mathbf{F}$$

$$\underline{\underline{P}} = \frac{1}{2}c_0\underline{\underline{C}}^T \underline{\underline{I}}_t \underline{\underline{I}} + \frac{1}{2}c_1\underline{\underline{C}} - \frac{1}{2}(3c_0 + c_1)\underline{\underline{I}} \quad \text{with} \quad \underline{\underline{C}} = \underline{\underline{F}}_t \underline{\underline{F}}$$

$$\boldsymbol{\sigma} = J^{-1}\mathbf{F} \cdot (\mathbf{F} \cdot \mathbf{P}^c)^c$$

$$\underline{\underline{\sigma}} = J^{-1}\underline{\underline{F}} \left( \underline{\underline{F}}_r \underline{\underline{P}}_t \right) = J^{-1}\underline{\underline{F}} \underline{\underline{F}}_r \underline{\underline{P}}_t$$

## Matrix/column notation

$$\delta \mathbf{P} = c_0 \mathbf{I}(\mathbf{F}^c : \delta \mathbf{F}) + \frac{1}{2} c_1 \{(\mathbf{F}^c \cdot \delta \mathbf{F})^c + (\mathbf{F}^c \cdot \delta \mathbf{F})\}$$

$$\begin{aligned} \delta \underline{\underline{P}}_{\approx} &= c_0 \underline{\underline{I}}_{\approx}^T \delta \underline{\underline{F}}_{\approx t} + \frac{1}{2} c_1 \left\{ (\underline{\underline{F}}_{\approx t} \delta \underline{\underline{F}}_{\approx})_r + (\underline{\underline{F}}_{\approx t} \delta \underline{\underline{F}}_{\approx}) \right\} \\ &= c_0 \underline{\underline{I}}_{\approx tc}^T \delta \underline{\underline{F}}_{\approx} + \frac{1}{2} c_1 \left( \underline{\underline{F}}_{\approx tr} \delta \underline{\underline{F}}_{\approx} + \underline{\underline{F}}_{\approx t} \delta \underline{\underline{F}}_{\approx} \right) \\ &= \underline{\underline{M}}_0 \delta \underline{\underline{F}}_{\approx} = \underline{\underline{M}}_0 \left( \underline{\underline{L}}_{\approx 0} \right)_t = \underline{\underline{M}}_0 \underline{\underline{F}}_{\approx tr} \underline{\underline{L}}_{\approx t} = \underline{\underline{M}}_1 \underline{\underline{L}}_{\approx t} \end{aligned}$$

$$\delta \boldsymbol{\sigma} = -\boldsymbol{\sigma}(\mathbf{I} : \mathbf{L}) + (\boldsymbol{\sigma}^c \cdot \mathbf{L}^c)^c + \boldsymbol{\sigma} \cdot \mathbf{L}^c + J^{-1} \mathbf{F} \cdot (\mathbf{F} \cdot \delta \mathbf{P}^c)^c$$

$$\begin{aligned} \delta \underline{\underline{\sigma}}_{\approx} &= -\underline{\underline{\sigma}}_{\approx}^T \underline{\underline{L}}_{\approx t} + \underline{\underline{\sigma}}_{\approx tr} \underline{\underline{L}}_{\approx t} + \underline{\underline{\sigma}}_{\approx} \underline{\underline{L}}_{\approx t} + J^{-1} \underline{\underline{F}}_{\approx} \underline{\underline{F}}_{\approx r} \delta \underline{\underline{P}}_{\approx t} \\ &= -\underline{\underline{\sigma}}_{\approx}^T \underline{\underline{L}}_{\approx t} + \underline{\underline{\sigma}}_{\approx tr} \underline{\underline{L}}_{\approx t} + \underline{\underline{\sigma}}_{\approx} \underline{\underline{L}}_{\approx t} + J^{-1} \underline{\underline{F}}_{\approx} \underline{\underline{F}}_{\approx rc} \delta \underline{\underline{P}}_{\approx} \\ &= \left[ -\underline{\underline{\sigma}}_{\approx}^T + \underline{\underline{\sigma}}_{\approx tr} + \underline{\underline{\sigma}}_{\approx} + J^{-1} \underline{\underline{F}}_{\approx} \underline{\underline{F}}_{\approx rc} \underline{\underline{M}}_1 \right] \underline{\underline{L}}_{\approx t} = \underline{\underline{M}} \underline{\underline{L}}_{\approx t} \end{aligned}$$

## $\sigma - \mathbf{A}$ model

- stress update
- consistent material stiffness tensor for  $\delta\sigma$   $\rightarrow$   
Updated Lagrange formulation



# Stress update

$$\boldsymbol{\sigma} = c_0 \text{tr}(\mathbf{A}) \mathbf{I} + c_1 \mathbf{A}$$

with  $\mathbf{A} = \frac{1}{2}(\mathbf{B} - \mathbf{I})$

$$= \frac{1}{2} c_0 \mathbf{B} : \mathbf{I} \mathbf{I} + \frac{1}{2} c_1 \mathbf{B} - \frac{1}{2} (3c_0 + c_1) \mathbf{I}$$

with  $\mathbf{B} = \mathbf{F} \cdot \mathbf{F}^c$

# Stiffness

$$\begin{aligned}\delta\sigma &= \frac{1}{2}c_0\delta\mathbf{B}:\mathbf{I}\mathbf{I} + \frac{1}{2}c_1\delta\mathbf{B} \\ &= \frac{1}{2}c_0\{(\mathbf{F}\cdot\delta\mathbf{F}^c)^c + \mathbf{F}\cdot\delta\mathbf{F}^c\}:\mathbf{I}\mathbf{I} + \frac{1}{2}c_1\{(\mathbf{F}\cdot\delta\mathbf{F}^c)^c + \mathbf{F}\cdot\delta\mathbf{F}^c\} \\ &= c_0(\mathbf{F}\cdot\delta\mathbf{F}^c):\mathbf{I}\mathbf{I} + \frac{1}{2}c_1\{(\mathbf{F}\cdot\delta\mathbf{F}^c)^c + \mathbf{F}\cdot\delta\mathbf{F}^c\} \\ &= c_0\mathbf{I}\mathbf{F}:\delta\mathbf{F}^c + \frac{1}{2}c_1\{(\mathbf{F}\cdot\delta\mathbf{F}^c)^c + \mathbf{F}\cdot\delta\mathbf{F}^c\} \\ &\quad \text{with} \quad \delta\mathbf{F} = \mathbf{L}\cdot\mathbf{F} = (\mathbf{F}^c\cdot\mathbf{L}^c)^c \quad \text{and} \quad \mathbf{L}^c = \vec{\nabla}\vec{u}\end{aligned}$$

## Matrix/column notation

$$\boldsymbol{\sigma} = \frac{1}{2}c_0 \mathbf{B} : \mathbf{I} \mathbf{I} + \frac{1}{2}c_1 \mathbf{B} - \frac{1}{2}(3c_0 + c_1) \mathbf{I} \quad \text{with} \quad \mathbf{B} = \mathbf{F} \cdot \mathbf{F}^c$$

$$\underline{\underline{\sigma}} = \frac{1}{2}c_0 \underline{\underline{B}}^T \underline{\underline{I}}_t \underline{\underline{I}} + \frac{1}{2}c_1 \underline{\underline{B}} - \frac{1}{2}(3c_0 + c_1) \underline{\underline{I}} \quad \text{with} \quad \underline{\underline{B}} = \underline{\underline{F}} \underline{\underline{F}}_t$$

$$\delta \boldsymbol{\sigma} = c_0 \mathbf{I} \mathbf{F} : \delta \mathbf{F}^c + \frac{1}{2}c_1 \{(\mathbf{F} \cdot \delta \mathbf{F}^c)^c + \mathbf{F} \cdot \delta \mathbf{F}^c\} \quad \text{with} \quad \delta \mathbf{F} = \mathbf{L} \cdot \mathbf{F} = (\mathbf{F}^c \cdot \mathbf{L}^c)^c$$

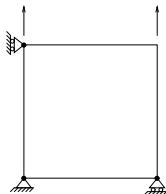
$$\begin{aligned} \delta \underline{\underline{\sigma}} &= c_0 \underline{\underline{I}} \underline{\underline{F}}^T \delta \underline{\underline{F}} + \frac{1}{2}c_1 \left\{ \underline{\underline{F}}_r \delta \underline{\underline{F}}_t + \underline{\underline{F}} \delta \underline{\underline{F}}_t \right\} \\ &= \left[ c_0 \underline{\underline{I}} \underline{\underline{F}}^T + \frac{1}{2}c_1 \left\{ \underline{\underline{F}}_{rc} + \underline{\underline{F}}_c \right\} \right] \delta \underline{\underline{F}} \quad \text{with} \quad \delta \underline{\underline{F}} = \left( \underline{\underline{F}}_t \underline{\underline{L}}_t \right)_r = \underline{\underline{F}}_{tr} \underline{\underline{L}}_t \end{aligned}$$

$$= \left[ c_0 \underline{\underline{I}} \underline{\underline{F}}^T \underline{\underline{F}}_{tr} + \frac{1}{2}c_1 \left( \underline{\underline{F}}_{rc} \underline{\underline{F}}_{tr} + \underline{\underline{F}}_c \underline{\underline{F}}_{tr} \right) \right] \underline{\underline{L}}_t = \underline{\underline{M}} \underline{\underline{L}}_t$$

# Examples

- Tensile test
- Shear test

# Tensile test



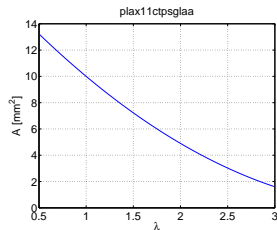
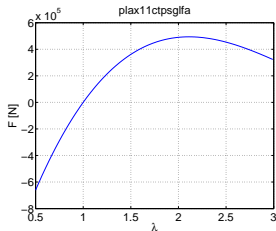
Cartesian			
initial width	$w_0$	100	mm
initial height	$h_0$	100	mm
initial thickness	$d_0$	0.1	mm

cylindrical			
initial radius	$r_0$	$\sqrt{(10/\pi)}$	mm
initial height	$h_0$	100	mm

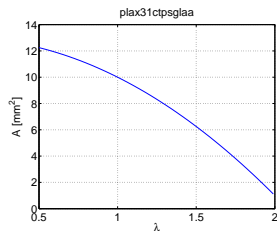
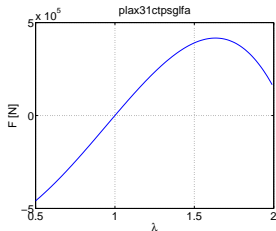
modulus	$C$	100000	MPa
Poisson ratio	$\nu$	0.3	-

# Elastic models in tensile test

plane stress;  $\sigma \sim \varepsilon$

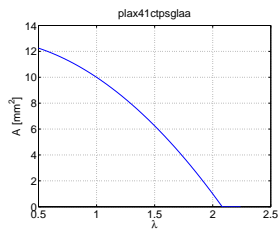
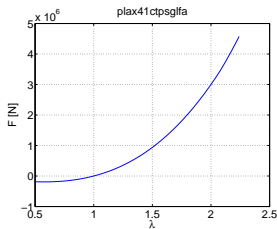


plane stress;  $\sigma \sim \mathbf{A}$



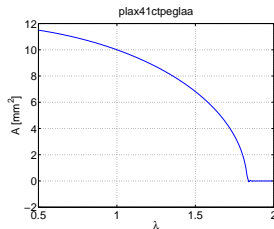
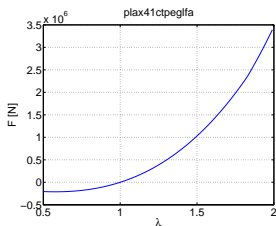
# Elastic models in tensile test

plane stress;  $\mathbf{P} \sim \mathbf{E}$

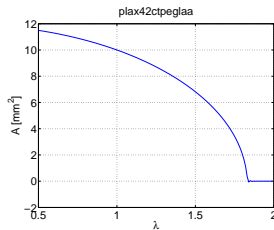
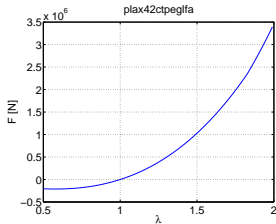


# Total Lagrange formulation

plane strain;  $\mathbf{P} \sim \mathbf{E}$ ; Upd.Lag.

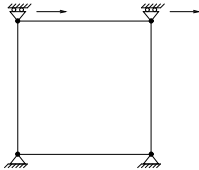


plane strain;  $\mathbf{P} \sim \mathbf{E}$ ; Tot.Lag.





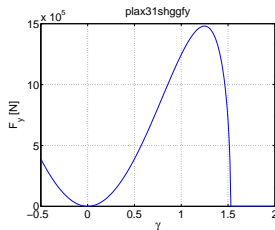
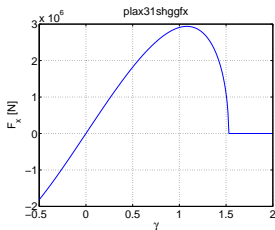
# Shear test



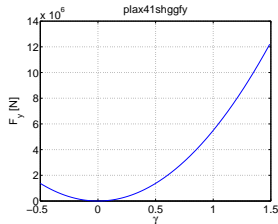
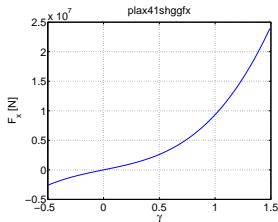
initial width	$w_0$	100	mm
initial height	$h_0$	100	mm
initial thickness	$d_0$	0.1	mm

# Elastic models in shear test

plane stress;  $\sigma \sim \mathbf{A}$



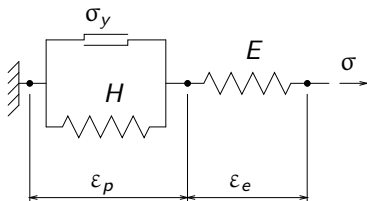
plane stress;  $\mathbf{P} \sim \mathbf{E}$



# ELASTOPLASTIC

[back to index](#)

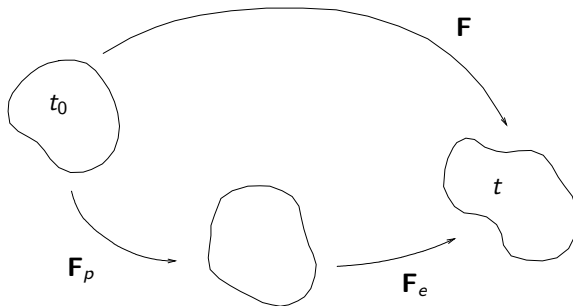
# Elastoplastic material behavior



$$\sigma < \sigma_y \rightarrow \epsilon = \epsilon_e$$

$$\sigma = \sigma_y \rightarrow \dot{\epsilon} = \dot{\epsilon}_e + \dot{\epsilon}_p$$

# Kinematics



$$\mathbf{F} = (\vec{\nabla}_0 \vec{X})^c = \mathbf{F}_e \cdot \mathbf{F}_p$$

$$\mathbf{C} = \mathbf{F}^c \cdot \mathbf{F} \quad ; \quad \mathbf{B} = \mathbf{F} \cdot \mathbf{F}^c \quad ; \quad \mathbf{E} = \frac{1}{2}(\mathbf{C} - \mathbf{I})$$

$$\mathbf{L} = \dot{\mathbf{F}} \cdot \mathbf{F}^{-1} = (\vec{\nabla} \vec{v})^c$$

$$= \mathbf{L}_e + \mathbf{L}_p = (\mathbf{D}_e + \boldsymbol{\Omega}_e) + (\mathbf{D}_p + \boldsymbol{\Omega}_p) = (\mathbf{D}_e + \boldsymbol{\Omega}_e) + \mathbf{D}_p$$

# Elastic deformation

metal alloys  $\rightarrow$  small elastic strains  $\rightarrow$  hypo-elastic model

$$\left. \begin{aligned} \boldsymbol{\sigma} &= {}^4\mathbf{C} : \boldsymbol{\Lambda}_e \\ {}^4\mathbf{C} &= c_0 \mathbb{I} + \frac{1}{2} c_1 ({}^4\mathbf{I} + {}^4\mathbf{I}^{rc}) = K \mathbb{I} + 2G \left( {}^4\mathbf{I} - \frac{1}{3} \mathbb{I} \right) \end{aligned} \right\}$$

invariant tensors

$$\begin{aligned} \boldsymbol{\sigma}_A &= \mathbf{A} \cdot \boldsymbol{\sigma} \cdot \mathbf{A}^c = \boldsymbol{\sigma}_A^* \quad \text{with} \quad \mathbf{A}^* = \mathbf{A} \cdot \mathbf{Q}^c \quad \forall \quad \mathbf{Q} \\ \dot{\boldsymbol{\sigma}}_A &= \mathbf{A} \cdot \left\{ (\mathbf{A}^{-1} \cdot \dot{\mathbf{A}}) \cdot \boldsymbol{\sigma} + \boldsymbol{\sigma} \cdot (\mathbf{A}^{-1} \cdot \dot{\mathbf{A}})^c + \dot{\boldsymbol{\sigma}} \right\} \cdot \mathbf{A}^c = \mathbf{A} \cdot \overset{\odot}{\boldsymbol{\sigma}}_A \cdot \mathbf{A}^c = \dot{\boldsymbol{\sigma}}_A^* \end{aligned}$$

objective elastic law

$$\overset{\odot}{\boldsymbol{\sigma}}_A = {}^4\mathbf{C} : \mathbf{D}_e$$

# Yield criterion and hardening

yield criterion

$$F = \bar{\sigma}^2 - \sigma_y^2(\bar{\varepsilon}_p)$$

effective plastic strain

$$\bar{\varepsilon}_p = \int_{\tau=0}^t \dot{\bar{\varepsilon}}_p d\tau$$

hardening law

$$\sigma_y = \sigma_y(\sigma_{y0}, \bar{\varepsilon}_p) \quad \text{with} \quad \frac{\partial \sigma_y}{\partial \bar{\varepsilon}_p} = H(\bar{\varepsilon}_p)$$

Kuhn-Tucker relations

$$\begin{aligned} \{(F < 0) \vee (F = 0 \wedge \dot{F} < 0)\} &\rightarrow \text{elastic} \\ \{(F = 0) \wedge (\dot{F} = 0)\} &\rightarrow \text{elastoplastic} \end{aligned}$$

# Von Mises plasticity

$$\bar{\sigma} = \sqrt{\frac{3}{2} \boldsymbol{\sigma}^d : \boldsymbol{\sigma}^d}$$

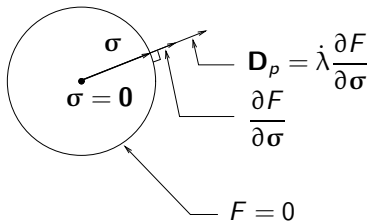
$$\dot{\bar{\epsilon}}_p = \sqrt{\frac{2}{3} \mathbf{D}_p : \mathbf{D}_p}$$

$$F = \frac{3}{2} \boldsymbol{\sigma}^d : \boldsymbol{\sigma}^d - \sigma_y^2(\bar{\epsilon}_p)$$

$$\begin{aligned} \dot{F} &= 2\bar{\sigma}\dot{\bar{\sigma}} - 2\sigma_y\dot{\sigma}_y = 2\bar{\sigma}\dot{\bar{\sigma}} - 2\sigma_y H \dot{\bar{\epsilon}}_p \\ &= 3\boldsymbol{\sigma}^d : \dot{\boldsymbol{\sigma}} - 2\sigma_y H \dot{\bar{\epsilon}}_p = 3\boldsymbol{\sigma}_A^d : \dot{\boldsymbol{\sigma}}_A - 2\sigma_y H \dot{\bar{\epsilon}}_p = 0 \end{aligned}$$



# Elastoplastic deformation



$$\mathbf{D}_p = \dot{\lambda} \frac{\partial F}{\partial \sigma} = \dot{\lambda} \mathbf{a}$$

$$\mathbf{a} = \frac{\partial F}{\partial \sigma^d} : \frac{\partial \sigma^d}{\partial \sigma} = [3\sigma^d : {}^4\mathbf{I}] : \frac{\partial}{\partial \sigma} \left\{ \sigma - \frac{1}{3} \text{tr}(\sigma) \mathbf{I} \right\} = 3\sigma^d : \left( {}^4\mathbf{I} - \frac{1}{3} \mathbf{I} \mathbf{I} \right) = 3\sigma^d$$

$$\dot{\varepsilon}_p = \dot{\lambda} \sqrt{\frac{2}{3} \mathbf{a} : \mathbf{a}}$$

# Constitutive model

$$\{(F < 0) \vee (F = 0 \wedge \dot{F} < 0)\} \rightarrow \mathbf{D} = \mathbf{D}_e \rightarrow \dot{\bar{\varepsilon}}_p = 0$$

$$\overset{\odot}{\sigma}_A = {}^4\mathbf{C} : \mathbf{D} \rightarrow \dot{\sigma}_A = {}^4\mathbf{C}_A : \mathbf{D}_A$$

$$\{(F = 0) \wedge (\dot{F} = 0)\} \rightarrow \mathbf{D} = \mathbf{D}_e + \mathbf{D}_p$$

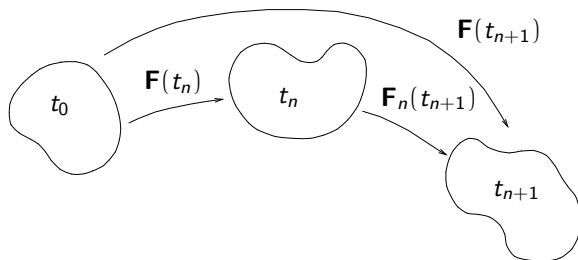
$$\left. \begin{array}{l} \overset{\odot}{\sigma}_A = {}^4\mathbf{C} : (\mathbf{D} - \dot{\lambda}\mathbf{a}) \\ 2\bar{\sigma}\dot{\bar{\sigma}} - 2\sigma_y H \dot{\bar{\varepsilon}}_p = 0 \end{array} \right\} \rightarrow$$

$$\left. \begin{array}{l} \dot{\sigma}_A = {}^4\mathbf{C}_A : (\mathbf{D}_A - \dot{\lambda}\mathbf{a}_A) \\ 3\sigma_A^d : \dot{\sigma}_A - 2\sigma_y H \dot{\lambda} \sqrt{\frac{2}{3}\mathbf{a}_A : \mathbf{a}_A} = 0 \end{array} \right\}$$

$$\left. \begin{array}{l} \dot{\sigma}_A = {}^4\mathbf{C}_A : (\mathbf{D}_A - \dot{\lambda}\mathbf{a}_A) \\ 3\sigma_A^d : {}^4\mathbf{C}_A : \mathbf{D}_A - \dot{\lambda} \left( 3\sigma_A^d : {}^4\mathbf{C}_A : \mathbf{a}_A + 2\sigma_y H \sqrt{\frac{2}{3}\mathbf{a}_A : \mathbf{a}_A} \right) = 0 \end{array} \right\}$$

$$\sigma_y = \sigma_y(\sigma_{y0}, \bar{\varepsilon}_p)$$

# Incremental analysis



$$\mathbf{F}(\tau) = \mathbf{F}_n(\tau) \cdot \mathbf{F}(t_n) \quad \rightarrow \quad \mathbf{F}_n(\tau) = (\vec{\nabla}_n \vec{x})^c = \mathbf{F}(\tau) \cdot \mathbf{F}^{-1}(t_n)$$

$$\mathbf{D} = \frac{1}{2} \left( \dot{\mathbf{F}}_n \cdot \mathbf{F}_n^{-1} + \mathbf{F}_n^{-c} \cdot \dot{\mathbf{F}}_n^c \right) = \frac{1}{2} \mathbf{R}_n \cdot \left( \dot{\mathbf{U}}_n \cdot \mathbf{U}_n^{-1} + \mathbf{U}_n^{-1} \cdot \dot{\mathbf{U}}_n \right) \cdot \mathbf{R}_n^c$$

$$\boldsymbol{\Omega} = \frac{1}{2} \left\{ \dot{\mathbf{F}}_n \cdot \mathbf{F}_n^{-1} - \mathbf{F}_n^{-c} \cdot \dot{\mathbf{F}}_n^c \right\} = \dot{\mathbf{R}}_n \cdot \mathbf{R}_n^c + \frac{1}{2} \mathbf{R}_n \cdot \left( \dot{\mathbf{U}}_n \cdot \mathbf{U}_n^{-1} - \mathbf{U}_n^{-1} \cdot \dot{\mathbf{U}}_n \right) \cdot \mathbf{R}_n^c$$

$$\mathbf{U}_n = \sum_{i=1}^3 \lambda_{ni} \vec{n}_{ni} \vec{n}_{ni} \quad ; \quad \boldsymbol{\Lambda}_n = \sum_{i=1}^3 \ln(\lambda_{ni}) \vec{n}_{ni} \vec{n}_{ni}$$

# Elastic stress predictor

elastic trial stress  $\boldsymbol{\sigma}_e = \boldsymbol{\sigma}(t_n) + {}^4\mathbf{C} : (\boldsymbol{\Lambda} - \boldsymbol{\Lambda}(t_n))$

yield criterion  $F = \frac{3}{2} \boldsymbol{\sigma}_e^d : \boldsymbol{\sigma}_e^d - \sigma_y^2(\sigma_{y0}, \bar{\varepsilon}_p(t_n))$

$$F \leq 0 \quad \rightarrow \quad \text{elastic increment}$$

$$F > 0 \quad \rightarrow \quad \text{elastoplastic increment}$$

matrix/column notation

$$\underline{\underline{\mathbf{C}}} = K \underline{\underline{\mathbf{I}}} \underline{\underline{\mathbf{I}}}^T + 2G \left( \underline{\underline{\mathbf{I}}} - \frac{1}{3} \underline{\underline{\mathbf{I}}} \underline{\underline{\mathbf{I}}}^T \right)$$

$$\underline{\underline{\Lambda}}_n \rightarrow \underline{\underline{\Lambda}}_n$$

$$\underline{\underline{\sigma}}_{D_e} = \underline{\underline{\sigma}}(t_n) + \underline{\underline{\mathbf{C}}} \underline{\underline{\Lambda}}_n \rightarrow \underline{\underline{\sigma}}_{D_e} \rightarrow$$

$$\underline{\underline{\sigma}}_e = \underline{\underline{R}}_n \underline{\underline{\sigma}}_{D_e} \underline{\underline{R}}_n^T$$

$$F = \frac{3}{2} \left( \underline{\underline{\sigma}}_{D_{tr}} \right)^T \left( \underline{\underline{\sigma}}_{D_{tr}} \right) - \sigma_y^2(\bar{\varepsilon}_p)$$

# Elastic increment

$$\boldsymbol{\sigma}(t_{n+1}) = \boldsymbol{\sigma}_e$$

$$\Delta\lambda = 0$$

$$\bar{\varepsilon}_p(t_{n+1}) = \bar{\varepsilon}_p(t_n)$$

$$\sigma_y(t_{n+1}) = \sigma_y(t_n)$$

## Elastoplastic increment

$$\left. \begin{aligned} \dot{\sigma}_A &= {}^4\mathbf{C}_A : (\mathbf{D}_A - \dot{\lambda} \mathbf{a}_A) \\ 3\sigma_A^d : {}^4\mathbf{C}_A : \mathbf{D}_A - \dot{\lambda} \left( 3\sigma_A^d : {}^4\mathbf{C}_A : \mathbf{a}_A + 2\sigma_v H \sqrt{\frac{2}{3} \mathbf{a}_A : \mathbf{a}_A} \right) &= 0 \end{aligned} \right\}$$

Dienes tensor and Dienes derivative

$$\left. \begin{aligned} \sigma_D &= \mathbf{R}_n^c \cdot \sigma \cdot \mathbf{R}_n \quad \rightarrow \quad \dot{\sigma}_D = \mathbf{R}_n^c \cdot \overset{\odot}{\sigma}_D \cdot \mathbf{R}_n \\ \mathbf{D}_D &= \mathbf{R}_n^c \cdot \mathbf{D} \cdot \mathbf{R}_n = \frac{1}{2} \left( \dot{\mathbf{U}}_n \cdot \mathbf{U}_n^{-1} + \mathbf{U}_n^{-1} \cdot \dot{\mathbf{U}}_n \right) \end{aligned} \right\}$$

$$\left. \begin{aligned} \dot{\sigma}_D &= {}^4\mathbf{C}_D : (\mathbf{D}_D - \dot{\lambda} \mathbf{a}_D) \\ 3\sigma_D^d : {}^4\mathbf{C}_D : \mathbf{D}_D - \dot{\lambda} \left( 3\sigma_D^d : {}^4\mathbf{C}_D : \mathbf{a}_D + 2\sigma_y H \sqrt{\frac{2}{3} \mathbf{a}_D : \mathbf{a}_D} \right) &= 0 \end{aligned} \right\}$$

# Rotation neutralized elastoplastic increment

- incremental rotation neutralized

$$\begin{aligned} t_n \leq \tau < t_{n+1} & : \quad \mathbf{R}_n = \mathbf{I} \quad ; \quad \mathbf{D}_D = \mathbf{D} \quad ; \quad \mathbf{a}_D = \mathbf{a} \quad ; \quad {}^4\mathbf{C}_D = {}^4\mathbf{C} \\ \tau = t_{n+1} & : \quad \mathbf{R}_n(t_{n+1}) = \mathbf{F}(t_{n+1}) \cdot \mathbf{U}^{-1}(t_{n+1}) \end{aligned}$$

- incremental principal strain directions constant  $\vec{n}_{ni}(\tau) = \vec{n}_{ni}(t_n)$

$$\mathbf{U}_n(\tau) = \sum_{i=1}^3 \lambda_{ni}(\tau) \vec{n}_{ni}(t_n) \vec{n}_{ni}(t_n)$$

$$\mathbf{D} = \dot{\mathbf{U}}_n \cdot \mathbf{U}_n^{-1} = \sum_{i=1}^3 \left( \frac{\dot{\lambda}_{ni}(\tau)}{\lambda_{ni}(\tau)} \right) \vec{n}_{ni}(t_n) \vec{n}_{ni}(t_n) = \dot{\boldsymbol{\Lambda}}_n$$

constitutive equations

$$\left. \begin{aligned} \dot{\boldsymbol{\sigma}}_D &= {}^4\mathbf{C} : \left\{ \dot{\boldsymbol{\Lambda}}_n - \dot{\lambda} \mathbf{a} \right\} \\ 3\boldsymbol{\sigma}_D^d : {}^4\mathbf{C} : \dot{\boldsymbol{\Lambda}}_n - \dot{\lambda} \left( 3\boldsymbol{\sigma}_D^d : {}^4\mathbf{C} : \mathbf{a} + 2\sigma_y H \sqrt{\frac{2}{3}} \mathbf{a} : \mathbf{a} \right) &= 0 \end{aligned} \right\}$$

## Rotation neutralized stress update

$$\left. \begin{aligned} \dot{\boldsymbol{\sigma}}_D &= {}^4\mathbf{C} : \left\{ \dot{\boldsymbol{\Lambda}}_n - \dot{\lambda} \mathbf{a} \right\} \\ 3\boldsymbol{\sigma}_D^d : {}^4\mathbf{C} : \dot{\boldsymbol{\Lambda}}_n - \dot{\lambda} \left( 3\boldsymbol{\sigma}_D^d : {}^4\mathbf{C} : \mathbf{a} + 2\sigma_y H \sqrt{\frac{2}{3} \mathbf{a} : \mathbf{a}} \right) &= 0 \end{aligned} \right\}$$

$$\left. \begin{aligned} \boldsymbol{\sigma}_D &= \boldsymbol{\sigma}_D(t_n) + {}^4\mathbf{C} : (\boldsymbol{\Lambda}_n - \Delta\lambda \mathbf{a}) \\ 3\boldsymbol{\sigma}_D^d : {}^4\mathbf{C} : \boldsymbol{\Lambda}_n - \Delta\lambda \left( 3\boldsymbol{\sigma}_D^d : {}^4\mathbf{C} : \mathbf{a} + 2\sigma_y H \sqrt{\frac{2}{3} \mathbf{a} : \mathbf{a}} \right) &= 0 \end{aligned} \right\}$$



# Iterative stress update

$$\left. \begin{aligned} {}^4\mathbf{R} : \delta\boldsymbol{\sigma}_D + \mathbf{t}\delta\lambda &= -\mathbf{s}_1 \\ \mathbf{u} : \delta\boldsymbol{\sigma}_D + \nu\delta\lambda &= -s_2 \end{aligned} \right\}$$

$${}^4\mathbf{R} = {}^4\mathbf{I} + 3\Delta\lambda {}^4\mathbf{C} : {}^4\mathbf{I}$$

$$\mathbf{t} = {}^4\mathbf{C} : \mathbf{a}$$

$$\mathbf{u} = (3 {}^4\mathbf{C} - \mathbb{I} : {}^4\mathbf{C}) : \boldsymbol{\Lambda}_n - \Delta\lambda \left\{ (3 {}^4\mathbf{C} - \mathbb{I} : {}^4\mathbf{C}) : \mathbf{a} + 4\sigma_y H \left( \frac{2}{3} \mathbf{a} : \mathbf{a} \right)^{-\frac{1}{2}} \mathbf{a} : {}^4\mathbf{I} \right\}$$

$$\nu = 3 {}^4\mathbf{C} : \mathbf{a} : \boldsymbol{\sigma}_D^d + 2\sigma_y H \sqrt{\frac{2}{3} \mathbf{a} : \mathbf{a}}$$

$$\mathbf{s}_1 = \boldsymbol{\sigma}_D - \boldsymbol{\sigma}_D(t_n) - {}^4\mathbf{C} : \boldsymbol{\Lambda}_n + \Delta\lambda {}^4\mathbf{C} : \mathbf{a}$$

$$s_2 = 3\boldsymbol{\sigma}_D^d : {}^4\mathbf{C} : \boldsymbol{\Lambda}_n - \Delta\lambda \left( 3\boldsymbol{\sigma}_D^d : {}^4\mathbf{C} : \mathbf{a} + 2\sigma_y H \sqrt{\frac{2}{3} \mathbf{a} : \mathbf{a}} \right)$$

# Stiffness

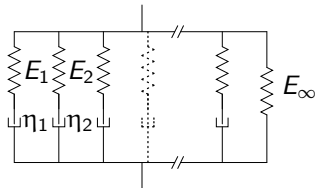
$$\left. \begin{aligned} \sigma_D - \sigma_D(t_n) - {}^4\mathbf{C} : \boldsymbol{\Lambda}_n + \Delta\lambda {}^4\mathbf{C} : \mathbf{a} &= 0 \\ 3\sigma_D^d : {}^4\mathbf{C} : \boldsymbol{\Lambda}_n - \Delta\lambda \left( 3\sigma_D^d : {}^4\mathbf{C} : \mathbf{a} + 2\sigma_y H \sqrt{\frac{2}{3} \mathbf{a} : \mathbf{a}} \right) &= 0 \end{aligned} \right\}$$

$$\left. \begin{aligned} \delta\sigma_D &= \sigma_D(t_n) + {}^4\mathbf{C} : \delta\boldsymbol{\Lambda}_n - \delta\lambda {}^4\mathbf{C} : \mathbf{a} - \Delta\lambda {}^4\mathbf{C} : \delta\mathbf{a} = 0 \\ 3\delta\sigma_D^d : {}^4\mathbf{C} : \boldsymbol{\Lambda}_n + 3\sigma_D^d : {}^4\mathbf{C} : \delta\boldsymbol{\Lambda}_n - \\ &\delta\lambda \left( 3\sigma_D^d : {}^4\mathbf{C} : \mathbf{a} + 2\sigma_y H \sqrt{\frac{2}{3} \mathbf{a} : \mathbf{a}} \right) - \\ &\Delta\lambda \left( 3\delta\sigma_D^d : {}^4\mathbf{C} : \mathbf{a} + 3\sigma_D^d : {}^4\mathbf{C} : \delta\mathbf{a} + \right. \\ &\quad \left. 2\delta\sigma_y H \sqrt{\frac{2}{3} \mathbf{a} : \mathbf{a}} + 2\sigma_y H \frac{1}{2} \left[ \frac{2}{3} \mathbf{a} : \mathbf{a} \right]^{-1/2} \frac{4}{3} \mathbf{a} : \delta\mathbf{a} \right) = 0 \end{aligned} \right\}$$

## LINEAR VISCOELASTIC

[back to index](#)

# Linear viscoelastic material behavior



$$\sigma(t) = \int_{\tau=0}^t {}^4\mathbf{C}(t-\tau) : \dot{\epsilon}(\tau) d\tau$$

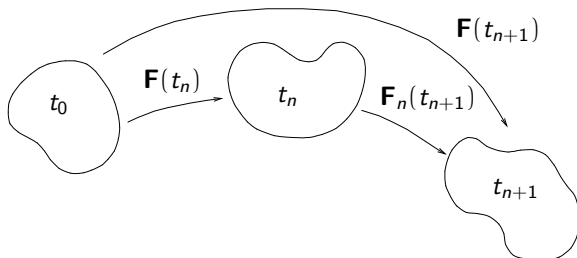
$${}^4\mathbf{C}(t) = {}^4\mathbf{C}_{\infty} + \sum_{i=1}^N {}^4\mathbf{C}_i e^{-\frac{t}{\tau_i}}$$

# Constitutive model

$$\left. \begin{aligned} \boldsymbol{\sigma}(t) &= \int_{\tau=0}^t {}^4\mathbf{C}(t-\tau) : \dot{\boldsymbol{\varepsilon}}(\tau) d\tau \\ {}^4\mathbf{C}(t) &= {}^4\mathbf{C}_{\infty} + \sum_{i=1}^N {}^4\mathbf{C}_i e^{-\frac{t}{\tau_i}} \end{aligned} \right\} \rightarrow$$

$$\begin{aligned} \boldsymbol{\sigma}(t) &= \int_{\tau=0}^t \left[ {}^4\mathbf{C}_{\infty} + \sum_{i=1}^N {}^4\mathbf{C}_i e^{-\frac{t-\tau}{\tau_i}} \right] : \dot{\boldsymbol{\varepsilon}}(\tau) d\tau \\ &= {}^4\mathbf{C}_{\infty} : \boldsymbol{\varepsilon}(t) + \sum_{i=1}^N {}^4\mathbf{C}_i : \int_{\tau=0}^t e^{-\frac{t-\tau}{\tau_i}} \dot{\boldsymbol{\varepsilon}}(\tau) d\tau \\ &= {}^4\mathbf{C}_{\infty} : \boldsymbol{\varepsilon}(t) + \sum_{i=1}^N \boldsymbol{\sigma}_i(t) \end{aligned}$$

# Incremental analysis



$$[0, t] \rightarrow [t_1 = 0, t_2, t_3, \dots, t_n, t_{n+1} = t]$$

$$\Delta t = t_{i+1} - t_i \quad ; \quad i = 1, \dots, n$$

$$\varepsilon(\tau) = \varepsilon(t_n) + (\tau - t_n) \frac{\Delta \varepsilon}{\Delta t} \rightarrow \dot{\varepsilon}(\tau) = \frac{\Delta \varepsilon}{\Delta t}$$

# Stress update

$$\begin{aligned}\sigma_i(t) &= {}^4\mathbf{C}_i : \int_{\tau=0}^t e^{-\frac{t-\tau}{\tau_i}} \dot{\epsilon}(\tau) d\tau \\&= {}^4\mathbf{C}_i : \left[ \int_{\tau=0}^{t_n} e^{-\frac{t-\tau}{\tau_i}} \dot{\epsilon}(\tau) d\tau + \int_{\tau=t_n}^t e^{-\frac{t-\tau}{\tau_i}} \dot{\epsilon}(\tau) d\tau \right] \\&= {}^4\mathbf{C}_i : \left[ e^{-\frac{\Delta t}{\tau_i}} \int_{\tau=0}^{t_n} e^{-\frac{t_n-\tau}{\tau_i}} \dot{\epsilon}(\tau) d\tau + \int_{\tau=t_n}^t e^{-\frac{t-\tau}{\tau_i}} \dot{\epsilon}(\tau) d\tau \right] \\&= e^{-\frac{\Delta t}{\tau_i}} {}^4\mathbf{C}_i : \int_{\tau=0}^{t_n} e^{-\frac{t_n-\tau}{\tau_i}} \dot{\epsilon}(\tau) d\tau + {}^4\mathbf{C}_i : \int_{\tau=t_n}^t e^{-\frac{t-\tau}{\tau_i}} \dot{\epsilon}(\tau) d\tau \\&= e^{-\frac{\Delta t}{\tau_i}} \sigma_i(t_n) + {}^4\mathbf{C}_i : \int_{\tau=t_n}^t e^{-\frac{t-\tau}{\tau_i}} \dot{\epsilon}(\tau) d\tau\end{aligned}$$

## Stress update

$$\begin{aligned}\sigma_i(t) &= e^{-\frac{\Delta t}{\tau_i}} \sigma_i(t_n) + {}^4\mathbf{C}_i : \int_{\tau=t_n}^t e^{-\frac{t-\tau}{\tau_i}} \frac{\Delta \boldsymbol{\varepsilon}}{\Delta t} d\tau = e^{-\frac{\Delta t}{\tau_i}} \sigma_i(t_n) + {}^4\mathbf{C}_i : \int_{\tau=t_n}^t e^{-\frac{t-\tau}{\tau_i}} d\tau \frac{\Delta \boldsymbol{\varepsilon}}{\Delta t} \\ &= e^{-\frac{\Delta t}{\tau_i}} \sigma_i(t_n) + {}^4\mathbf{C}_i : \tau_i \left(1 - e^{-\frac{\Delta t}{\tau_i}}\right) \frac{\Delta \boldsymbol{\varepsilon}}{\Delta t}\end{aligned}$$

$$\begin{aligned}\boldsymbol{\sigma}(t) &= {}^4\mathbf{C}_\infty : \boldsymbol{\varepsilon}(t) + \sum_{i=1}^N \sigma_i(t) \\ &= {}^4\mathbf{C}_\infty : \boldsymbol{\varepsilon}(t) + \sum_{i=1}^N \left[ e^{-\frac{\Delta t}{\tau_i}} \sigma_i(t_n) + {}^4\mathbf{C}_i : \tau_i \left(1 - e^{-\frac{\Delta t}{\tau_i}}\right) \frac{\Delta \boldsymbol{\varepsilon}}{\Delta t} \right]\end{aligned}$$



# Stiffness

$$\boldsymbol{\sigma}(t) = {}^4\mathbf{C}_{\infty} : \boldsymbol{\varepsilon}(t) + \sum_{i=1}^N \left[ e^{-\frac{\Delta t}{\tau_i}} \boldsymbol{\sigma}_i(t_n) + {}^4\mathbf{C}_i : \frac{\Delta \boldsymbol{\varepsilon}}{\Delta t} \tau_i \left( 1 - e^{-\frac{\Delta t}{\tau_i}} \right) \right]$$

$$\begin{aligned} \delta \boldsymbol{\sigma} &= \left[ {}^4\mathbf{C}_{\infty} + \sum_{i=1}^N {}^4\mathbf{C}_i \frac{\tau_i}{\Delta t} \left( 1 - e^{-\frac{\Delta t}{\tau_i}} \right) \right] : \delta \boldsymbol{\varepsilon} \\ &= {}^4\mathbf{M} : \delta \boldsymbol{\varepsilon} \end{aligned}$$

# Isotropic material

$$\begin{aligned}\boldsymbol{\sigma} &= {}^4\mathbf{C} : \boldsymbol{\varepsilon} \\ &= \left[ \lambda \mathbf{I} \mathbf{I} + 2\mu {}^4\mathbf{I}^s \right] : \boldsymbol{\varepsilon} = \left[ \lambda \mathbf{I} \mathbf{I} + \mu \left( {}^4\mathbf{I} + {}^4\mathbf{I}^{rc} \right) \right] : \boldsymbol{\varepsilon} = \lambda \mathbf{I} \operatorname{tr}(\boldsymbol{\varepsilon}) + 2\mu \boldsymbol{\varepsilon} \\ &= (3\lambda + 2\mu) \frac{1}{3} \operatorname{tr}(\boldsymbol{\varepsilon}) \mathbf{I} + 2\mu \boldsymbol{\varepsilon}^d = (3\lambda + 2\mu) \boldsymbol{\varepsilon}^h + 2\mu \boldsymbol{\varepsilon}^d \\ &= 3K \boldsymbol{\varepsilon}^h + 2G \boldsymbol{\varepsilon}^d \\ &= \boldsymbol{\sigma}^h + \boldsymbol{\sigma}^d\end{aligned}$$

$$K = \frac{1}{3} (3\lambda + 2\mu) = \frac{E}{3(1-2\nu)}$$

$$\mu = G = \frac{E}{2(1+\nu)}$$

$$\lambda = \frac{E\nu}{(1+\nu)(1-2\nu)}$$

# Isotropic viscoelastic material

$$\begin{aligned}\boldsymbol{\sigma}(t) &= \boldsymbol{\sigma}^h(t) + \boldsymbol{\sigma}^d(t) \\ &= 3 \int_{\tau=0}^t K(t-\tau) \frac{d}{d\tau} \{\boldsymbol{\varepsilon}^h(\tau)\} d\tau + 2 \int_{\tau=0}^t G(t-\tau) \frac{d}{d\tau} \{\boldsymbol{\varepsilon}^d(\tau)\} d\tau\end{aligned}$$

$$K(t) = K_{\infty} + \sum_{i=1}^n K_i e^{-\frac{t}{\tau_i}} = \frac{1}{3(1-2\nu)} \left[ E_{\infty} + \sum_{i=1}^n E_i e^{-\frac{t}{\tau_i}} \right]$$

$$G(t) = G_{\infty} + \sum_{i=1}^n G_i e^{-\frac{t}{\tau_i}} = \frac{1}{2(1+\nu)} \left[ E_{\infty} + \sum_{i=1}^n E_i e^{-\frac{t}{\tau_i}} \right]$$

## Stress update

$$\begin{aligned}\boldsymbol{\sigma}(t) &= {}^4\mathbf{C}_\infty : \boldsymbol{\varepsilon}(t) + \sum_{i=1}^N \boldsymbol{\sigma}_i(t) \\ &= {}^4\mathbf{C}_\infty : \boldsymbol{\varepsilon}(t) + \sum_{i=1}^N \left[ e^{-\frac{\Delta t}{\tau_i}} \boldsymbol{\sigma}_i(t_n) + {}^4\mathbf{C}_i : \tau_i \left( 1 - e^{-\frac{\Delta t}{\tau_i}} \right) \frac{\Delta \boldsymbol{\varepsilon}}{\Delta t} \right] \\ &= 3K_\infty \Delta \boldsymbol{\varepsilon}^h + 2G_\infty \Delta \boldsymbol{\varepsilon}^d + \\ &\quad \sum_{i=1}^N \left[ e^{-\frac{\Delta t}{\tau_i}} \boldsymbol{\sigma}_i(t_n) + \frac{\tau_i}{\Delta t} \left( 1 - e^{-\frac{\Delta t}{\tau_i}} \right) \{ 3K_i \Delta \boldsymbol{\varepsilon}^h + 2G_i \Delta \boldsymbol{\varepsilon}^d \} \right]\end{aligned}$$

# Stiffness

$$\delta \boldsymbol{\sigma} = 3K_{\infty} \delta \boldsymbol{\epsilon}^h + 2G_{\infty} \delta \boldsymbol{\epsilon}^d + \sum_{i=1}^N \frac{\tau_i}{\Delta t} \left( 1 - e^{-\frac{\Delta t}{\tau_i}} \right) \{ 3K_i \delta \boldsymbol{\epsilon}^h + 2G_i \delta \boldsymbol{\epsilon}^d \}$$

## Matrix/column notation

$$\begin{aligned} \underline{\underline{\tilde{g}}}(t) = & \left( 3K_{\infty} \underline{\underline{A}}^h + 2G_{\infty} \underline{\underline{A}}^d \right) \Delta_{\tilde{\xi}} + \\ & \sum_{i=1}^N \left[ e^{-\frac{\Delta t}{\tau_i}} \underline{\underline{g}}_i(t_n) + \right. \\ & \left. \frac{\tau_i}{\Delta t} \left( 1 - e^{-\frac{\Delta t}{\tau_i}} \right) \left\{ 3K_i \underline{\underline{A}}^h + 2G_i \underline{\underline{A}}^d \right\} \right] \Delta_{\tilde{\xi}} \end{aligned}$$

$$\begin{aligned} \delta \underline{\underline{\tilde{g}}}(t) = & \left[ \left( 3K_{\infty} \underline{\underline{A}}^h + 2G_{\infty} \underline{\underline{A}}^d \right) \delta_{\tilde{\xi}} + \right. \\ & \left. \sum_{i=1}^N \frac{\tau_i}{\Delta t} \left( 1 - e^{-\frac{\Delta t}{\tau_i}} \right) \left( 3K_i \underline{\underline{A}}^h + 2G_i \underline{\underline{A}}^d \right) \right] \delta_{\tilde{\xi}} \end{aligned}$$

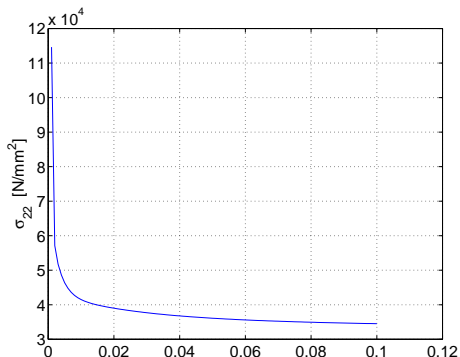
## Initial stiffness formulation

$$\Delta \boldsymbol{\sigma}(t) = 3K_0 \Delta \boldsymbol{\varepsilon}^h + 2G_0 \Delta \boldsymbol{\varepsilon}^d - \sum_{i=1}^N \left[ 1 - \left( 1 - e^{-\frac{\Delta t}{\tau_i}} \right) \frac{\tau_i}{\Delta t} \right] \{ 3K_i \Delta \boldsymbol{\varepsilon}^h + 2G_i \Delta \boldsymbol{\varepsilon}^d \} - \sum_{i=1}^N \left( 1 - e^{-\frac{\Delta t}{\tau_i}} \right) \{ \boldsymbol{\sigma}_i^h(t_n) + \boldsymbol{\sigma}_i^d(t_n) \}$$

$$\delta \boldsymbol{\sigma} = 3K_0 \delta \boldsymbol{\varepsilon}^h + 2G_0 \delta \boldsymbol{\varepsilon}^d - \sum_{i=1}^N \left[ 1 - \left( 1 - e^{-\frac{\Delta t}{\tau_i}} \right) \frac{\tau_i}{\Delta t} \right] \{ 3K_i \delta \boldsymbol{\varepsilon}^h + 2G_i \delta \boldsymbol{\varepsilon}^d \}$$

# Tensile test

	$E$ [MPa]	$\tau$ [s]		$E$ [MPa]	$\tau$ [s]
1	3.0e6	3.1e-8	2	1.4e6	3.0e-7
3	3.9e6	3.0e-6	4	5.4e6	2.9e-5
5	1.3e6	2.8e-4	6	2.3e5	2.7e-3
7	7.6e4	2.6e-2	8	3.7e4	2.5e-1
9	3.3e4	2.5e+0	10	1.7e4	2.4e+1
11	8.0e3	2.3e+2	12	1.2e4	2.2e+3

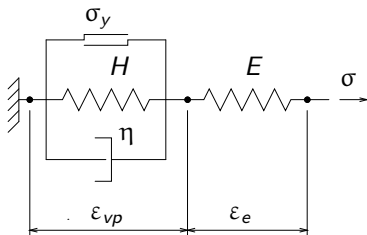




# VISCOPLASTIC

[back to index](#)

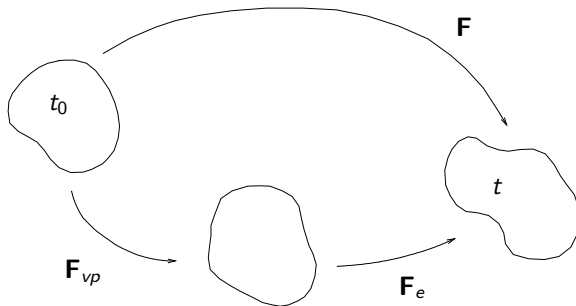
# Viscoplastic material behavior



$$\sigma < \sigma_y \rightarrow \epsilon = \epsilon_e$$

$$\sigma \geq \sigma_y \rightarrow \dot{\epsilon} = \dot{\epsilon}_e + \dot{\epsilon}_{vp}$$

# Kinematics



$$\mathbf{F} = (\vec{\nabla}_0 \vec{X})^c = \mathbf{F}_e \cdot \mathbf{F}_{vp}$$

$$\mathbf{C} = \mathbf{F}^c \cdot \mathbf{F} \quad ; \quad \mathbf{B} = \mathbf{F} \cdot \mathbf{F}^c \quad ; \quad \mathbf{E} = \frac{1}{2}(\mathbf{C} - \mathbf{I})$$

$$\mathbf{L} = \dot{\mathbf{F}} \cdot \mathbf{F}^{-1} = (\vec{\nabla} \vec{v})^c$$

$$= \mathbf{L}_e + \mathbf{L}_{vp} = (\mathbf{D}_e + \boldsymbol{\Omega}_e) + (\mathbf{D}_{vp} + \boldsymbol{\Omega}_{vp}) = (\mathbf{D}_e + \boldsymbol{\Omega}_e) + \mathbf{D}_{vp}$$

# Elastic deformation

polymers → large elastic strains → hyper-elastic model

$$\left. \begin{aligned} \mathbf{P} &= \frac{\partial W(\mathbf{E}_e)}{\partial \mathbf{E}_e} = 2 \frac{\partial W}{\partial \mathbf{C}_e} = \mathbf{F}^{-1} \cdot \boldsymbol{\tau} \cdot \mathbf{F}^{-c} \rightarrow \dot{\mathbf{P}} = 2 \frac{\partial^2 W}{\partial \mathbf{C}^2} : \dot{\mathbf{C}} \\ W(\lambda_1, \lambda_2, \lambda_3) &= \frac{1}{2} \mu \{ \lambda_1^2 + \lambda_2^2 + \lambda_3^2 - 3 - 2 \ln(J) \} + \frac{1}{2} \lambda \{ \ln(J) \}^2 \\ \text{with } \lambda &= \frac{\nu E}{(1 + \nu)(1 - 2\nu)} \quad ; \quad \mu = \frac{E}{2(1 + \nu)} \end{aligned} \right\}$$

# Yield criterion and hardening

yield criterion

$$F = \bar{\tau} - \tau_y(\bar{\varepsilon}_{vp})$$

effective viscoplastic strain

$$\bar{\varepsilon}_{vp} = \int_{\tau=0}^t \dot{\bar{\varepsilon}}_{vp} d\tau$$

hardening law

$$\tau_y = \tau_y(\tau_{y0}, \bar{\varepsilon}_{vp}) \quad \text{with} \quad \frac{\partial \tau_y}{\partial \bar{\varepsilon}_p} = H(\bar{\varepsilon}_p)$$

Kuhn-Tucker relations

$$\begin{array}{ll} F < 0 & \rightarrow \text{elastic deformation} \\ F \geq 0 & \rightarrow \text{viscoplastic deformation} \end{array}$$

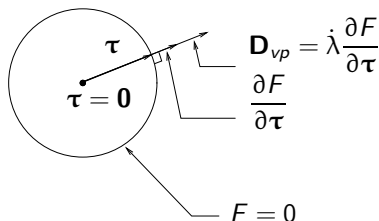
# Von Mises plasticity

$$\bar{\tau} = \sqrt{\frac{3}{2} \boldsymbol{\tau}^d : \boldsymbol{\tau}^d}$$

$$\dot{\bar{\epsilon}}_{vp} = \sqrt{\frac{2}{3} \mathbf{D}_{vp} : \mathbf{D}_{vp}}$$

$$F = \sqrt{\frac{3}{2} \boldsymbol{\tau}^d : \boldsymbol{\tau}^d} - \tau_y(\bar{\epsilon}_{vp})$$

# Viscoplastic deformation



$$\mathbf{D}_{vp} = \dot{\lambda} \frac{\partial F}{\partial \tau} = \dot{\lambda} \mathbf{a} \quad \rightarrow \quad \dot{\mathbf{C}}_{vp} = 2 \mathbf{F}^c \cdot \mathbf{D}_{vp} \cdot \mathbf{F} = 2 \dot{\lambda} \mathbf{F}^c \cdot \mathbf{a} \cdot \mathbf{F}$$

$$\begin{aligned} \mathbf{a} &= \frac{\partial F}{\partial \tau^d} : \frac{\partial \tau^d}{\partial \tau} = \left[ \frac{3}{2} \left( \frac{3}{2} \tau^d : \tau^d \right)^{-1/2} \tau^d : {}^4\mathbf{I} \right] : \left[ \frac{\partial}{\partial \tau} \left\{ \tau - \frac{1}{3} \text{tr}(\tau) \mathbf{I} \right\} \right] \\ &= \frac{3}{2} \left( \frac{3}{2} \tau^d : \tau^d \right)^{-1/2} \tau^d : \left( {}^4\mathbf{I} - \frac{1}{3} \mathbf{I} \mathbf{I} \right) = \frac{3}{2} \left( \frac{3}{2} \tau^d : \tau^d \right)^{-1/2} \tau^d \\ &= \frac{3}{2} \frac{1}{\bar{\tau}} \tau^d \end{aligned}$$

$$\dot{\varepsilon}_{vp} = \dot{\lambda} \sqrt{\frac{2}{3} \mathbf{a} : \mathbf{a}}$$

# Constitutive model

$$F < 0 \quad \rightarrow \quad \dot{\mathbf{C}} = \dot{\mathbf{C}}_e$$

$$\dot{\mathbf{P}} = 2 \frac{\partial^2 W}{\partial \mathbf{C}^2} : \dot{\mathbf{C}} \quad ; \quad \dot{\mathbf{C}}_{vp} = \mathbf{0} \quad ; \quad \dot{\bar{\epsilon}}_{vp} = 0$$

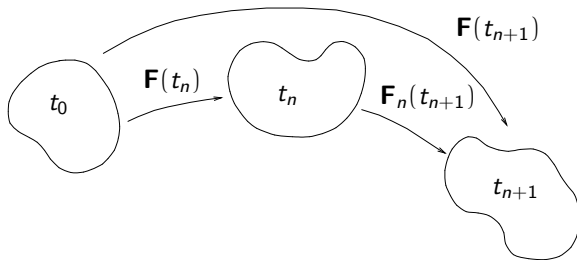
$$F \geq 0 \quad \rightarrow \quad \dot{\mathbf{C}} = \dot{\mathbf{C}}_e + \dot{\mathbf{C}}_{vp} \quad \rightarrow$$

$$\left. \begin{aligned} \dot{\mathbf{P}} &= 2 \frac{\partial^2 W}{\partial \mathbf{C}^2} : \left( \dot{\mathbf{C}} - 2 \mathbf{F}^c \cdot \dot{\lambda} \mathbf{a} \cdot \mathbf{F} \right) \\ \dot{\lambda} &= \gamma \phi(F) = \gamma \left( \frac{F}{\tau_{y0}} \right)^N \end{aligned} \right\}$$

$$\tau_y = \tau_y(\tau_{y0}, \bar{\epsilon}_{vp}) \quad ; \quad \dot{\bar{\epsilon}}_{vp} = \dot{\lambda} \sqrt{\frac{2}{3} \mathbf{a} : \mathbf{a}}$$



# Incremental analysis



$$\mathbf{F}(\tau) = \mathbf{F}_n(\tau) \cdot \mathbf{F}(t_n) \quad \rightarrow \quad \mathbf{F}_n(\tau) = \mathbf{F}(\tau) \cdot \mathbf{F}^{-1}(t_n)$$

$$\mathbf{F}_n = (\vec{\nabla}_n \vec{x})^c = \mathbf{R}_n \cdot \mathbf{U}_n \quad ; \quad J_n = \det(\mathbf{F}_n) \quad ; \quad \vec{\nabla} = \mathbf{F}_n^{-c} \cdot \vec{\nabla}_n$$

$$\mathbf{D} = \frac{1}{2} \left\{ (\vec{\nabla} \vec{v})^c + (\vec{\nabla} \vec{v}) \right\} = \frac{1}{2} \left( \dot{\mathbf{F}}_n \cdot \mathbf{F}_n^{-1} + \mathbf{F}_n^{-c} \cdot \dot{\mathbf{F}}_n^c \right)$$

# Elastic stress predictor

elastic trial stress  $\mathbf{P}_e = \mathbf{P}_n + 2 \frac{\partial^2 W}{\partial \mathbf{G}^2} : (\mathbf{C} - \mathbf{C}(t_n)) \rightarrow \boldsymbol{\tau}_e = \mathbf{F} \cdot \mathbf{P}_e \cdot \mathbf{F}^c$

yield criterion  $F = \sqrt{\frac{3}{2} (\boldsymbol{\tau}_e)^d : (\boldsymbol{\tau}_e)^d} - \tau_y(\tau_{y0}, \bar{\epsilon}_{vp}(t_n))$

$F < 0 \rightarrow$  elastic increment

$F \geq 0 \rightarrow$  elastoviscoplastic increment

matrix/column notation

$$\underline{\underline{\tau}}_e = \underline{\underline{A}} + \underline{\underline{H}}_c \underline{e}_n$$

$$F = \sqrt{\frac{3}{2} \left( \underline{\underline{\tau}}_e \right)^T \left( \underline{\underline{\tau}}_e \right)_t} - \zeta(\kappa)$$

with 
$$\begin{cases} \underline{\underline{H}} = 2\{\mu - \lambda \ln(J)\} \underline{\underline{I}} + \lambda \underline{\underline{I}} \underline{\underline{I}}^T \\ \underline{e}_n = \frac{1}{2} \left( \underline{\underline{I}} - \underline{\underline{F}}_n^{-T} \underline{\underline{F}}_n^{-1} \right) \rightarrow \underline{e}_n \\ \underline{\underline{A}} = \underline{\underline{F}}_n \underline{\underline{\tau}}(t_n) \underline{\underline{F}}_n^T \rightarrow \underline{\underline{A}} \end{cases}$$

# Elastic increment

$$\boldsymbol{\tau}(t_{n+1}) = \boldsymbol{\tau}_e$$

$$\Delta\lambda = 0$$

$$\bar{\varepsilon}_{vp}(t_{n+1}) = \bar{\varepsilon}_{vp}(t_n)$$

$$\tau_y(t_{n+1}) = \tau_y(t_n)$$

# Viscoplastic increment

$$\left. \begin{aligned} \dot{\mathbf{P}} &= 2 \frac{\partial^2 W}{\partial \mathbf{C}^2} : (\dot{\mathbf{C}} - 2 \mathbf{F}^c \cdot \dot{\lambda} \mathbf{a} \cdot \mathbf{F}) \\ \dot{\lambda} &= \gamma \phi(F) = \gamma \left( \frac{F}{\tau_{y0}} \right)^N \end{aligned} \right\}$$

$$\left. \begin{aligned} \mathbf{P} &= \mathbf{P}(t_n) + 2 \frac{\partial^2 W}{\partial \mathbf{C}^2} : \{ \mathbf{C} - \mathbf{C}(t_n) - 2 \mathbf{F}^c \cdot \Delta \lambda \mathbf{a} \cdot \mathbf{F} \} \\ \Delta \lambda &= \Delta t \gamma \phi(F) \end{aligned} \right\}$$

## Viscoplastic increment

$$\left. \begin{aligned} \mathbf{F}^{-1} \cdot \boldsymbol{\tau} \cdot \mathbf{F}^{-c} &= \mathbf{F}^{-1}(t_n) \cdot \boldsymbol{\tau}(t_n) \cdot \mathbf{F}^{-c}(t_n) + 2 \frac{\partial^2 W}{\partial \mathbf{C}^2} : \{ \mathbf{C} - \mathbf{C}(t_n) - 2 \mathbf{F}^c \cdot \Delta \lambda \mathbf{a} \cdot \mathbf{F} \} \\ \Delta \lambda &= \Delta t \gamma \phi(F) \end{aligned} \right\}$$

$$\mathbf{F}_n = \mathbf{F} \cdot \mathbf{F}^{-1}(t_n) \quad \rightarrow \quad \mathbf{C} - \mathbf{C}(t_n) = \mathbf{F}^c \cdot (\mathbf{I} - \mathbf{F}_n^{-c} \cdot \mathbf{F}_n^{-1}) \cdot \mathbf{F} = 2 \mathbf{F}^c \cdot \mathbf{e}_n \cdot \mathbf{F}$$

$$\left. \begin{aligned} \boldsymbol{\tau} &= \mathbf{F}_n \cdot \boldsymbol{\tau}(t_n) \cdot \mathbf{F}_n^c + 4 \mathbf{F} \cdot \frac{\partial^2 W}{\partial \mathbf{C}^2} : \mathbf{F}^c \cdot (\mathbf{e}_n - \Delta \lambda \mathbf{a}) \cdot \mathbf{F} \cdot \mathbf{F}^c \\ \Delta \lambda &= \Delta t \gamma \phi(F) \end{aligned} \right\}$$

$${}^4\mathbf{H} = 4 \mathbf{F} \cdot \left( \mathbf{F} \cdot \frac{\partial^2 W}{\partial \mathbf{C}^2} \cdot \mathbf{F}^c \right)^{lc,rc} \cdot \mathbf{F}^c = 2 \{ \mu - \lambda \ln(J) \} {}^4\mathbf{I}^{rc} + \lambda \mathbf{I}$$

$$\left. \begin{aligned} \boldsymbol{\tau} &= \mathbf{F}_n \cdot \boldsymbol{\tau}(t_n) \cdot \mathbf{F}_n^c + {}^4\mathbf{H} : (\mathbf{e}_n - \Delta \lambda \mathbf{a}) = \boldsymbol{\tau}_e - \Delta \lambda {}^4\mathbf{H} : \mathbf{a} \\ \Delta \lambda &= \Delta t \gamma \phi(F) \end{aligned} \right\}$$

## Iterative stress update

$$\left. \begin{aligned} \boldsymbol{\tau} - \boldsymbol{\tau}_e + \Delta\lambda {}^4\mathbf{H} : \mathbf{a} &= \mathbf{0} \\ \Delta\lambda - \Delta t \gamma \phi(F) &= 0 \end{aligned} \right\}$$

$$\left. \begin{aligned} \delta\boldsymbol{\tau} - \delta\boldsymbol{\tau}_e + {}^4\mathbf{H} : \mathbf{a} \delta\lambda + \Delta\lambda \delta {}^4\mathbf{H} : \mathbf{a} + \Delta\lambda {}^4\mathbf{H} : \delta\mathbf{a} &= -\mathbf{s}_1 \\ \delta\lambda - \Delta t \gamma \left( \frac{\partial \phi}{\partial F} \right) \mathbf{a} : \delta\boldsymbol{\tau} - \Delta t \gamma \left( \frac{\partial \phi}{\partial F} \right) \left( \frac{\partial F}{\partial \bar{\epsilon}_{vp}} \right) \delta\lambda &= -s_2 \end{aligned} \right\}$$

$$\text{with } \left\{ \begin{aligned} \delta\boldsymbol{\tau}_e &= \mathbf{M}_1 \delta\lambda + {}^4\mathbf{M}_2 : \delta\boldsymbol{\tau} \\ \delta {}^4\mathbf{H} &= \left( \frac{\partial {}^4\mathbf{H}}{\partial J} \right) \delta J = {}^4\mathbf{c} \delta J \\ \delta\mathbf{a} &= \left( \frac{\partial \mathbf{a}}{\partial \boldsymbol{\tau}} \right) : \delta\boldsymbol{\tau} = {}^4\mathbf{b} : \delta\boldsymbol{\tau} \\ \delta J &= J_1 \delta\lambda + \mathbf{J}_2 : \delta\boldsymbol{\tau} \end{aligned} \right.$$

plane strain

$$\delta\boldsymbol{\tau}_{tr} = \mathbf{0} \quad ; \quad \delta J = 0$$

# Iterative stress update

$$\left. \begin{aligned} {}^4\mathbf{R} : \delta\boldsymbol{\tau} + \mathbf{t} \delta\lambda &= -\mathbf{s}_1 \\ \mathbf{u} : \delta\boldsymbol{\tau} + v \delta\lambda &= -s_2 \end{aligned} \right\}$$

$${}^4\mathbf{R} = {}^4\mathbf{I} + \Delta\lambda {}^4\mathbf{H} : {}^4\mathbf{b} + \Delta\lambda {}^4\mathbf{c} : \mathbf{a} \mathbf{J}_2 - {}^4\mathbf{M}_2$$

$$\mathbf{t} = {}^4\mathbf{H} : \mathbf{a} + \Delta\lambda {}^4\mathbf{c} : \mathbf{a} \mathbf{J}_1 - \mathbf{M}_1$$

$$\mathbf{u} = -\Delta t \gamma \left( \frac{\partial \phi}{\partial F} \right) \mathbf{a}$$

$$v = 1 - \Delta t \gamma \left( \frac{\partial \phi}{\partial F} \right) \left( \frac{\partial F}{\partial \bar{\varepsilon}_{vp}} \right)$$

$$\mathbf{s}_1 = \boldsymbol{\tau} - \boldsymbol{\tau}_e + \Delta\lambda {}^4\mathbf{H} : \mathbf{a}$$

$$s_2 = \Delta\lambda - \Delta t \gamma \phi(F)$$





# Stiffness

$$\left. \begin{aligned} \boldsymbol{\tau} &= \mathbf{F}_n \cdot \boldsymbol{\tau}(t_n) \cdot \mathbf{F}_n^c + {}^4\mathbf{H} : \mathbf{e}_n - \Delta\lambda {}^4\mathbf{H} : \mathbf{a} \\ \Delta\lambda &= \Delta t \gamma \phi(F) \end{aligned} \right\} \rightarrow$$

$$\left. \begin{aligned} \delta\boldsymbol{\tau} &= \delta\mathbf{F}_n \cdot \boldsymbol{\tau}(t_n) \cdot \mathbf{F}_n^c + \mathbf{F}_n \cdot \boldsymbol{\tau}(t_n) \cdot \delta\mathbf{F}_n^c + \delta {}^4\mathbf{H} : (\mathbf{e}_n - \Delta\lambda \mathbf{a}) + \\ &\quad {}^4\mathbf{H} : \delta\mathbf{e}_n - {}^4\mathbf{H} : \mathbf{a} \delta\lambda - \Delta\lambda {}^4\mathbf{H} : \left( \frac{\partial \mathbf{a}}{\partial \boldsymbol{\tau}} \right) : \delta\boldsymbol{\tau} \\ \delta\lambda &= \left[ \left\{ \Delta t \gamma \left( \frac{\partial \phi}{\partial F} \right) \right\} / \left\{ 1 - \Delta t \gamma \left( \frac{\partial \phi}{\partial F} \right) \left( \frac{\partial F}{\partial \varepsilon_{vp}} \right) \right\} \right] \mathbf{a} : \delta\boldsymbol{\tau} = c_1 \mathbf{a} : \delta\boldsymbol{\tau} \end{aligned} \right\}$$

$$\left\{ {}^4\mathbf{I} + \Delta\lambda {}^4\mathbf{H} : \left( \frac{\partial \mathbf{a}}{\partial \boldsymbol{\tau}} \right) + c_1 {}^4\mathbf{H} : \mathbf{a} \mathbf{a} \right\} : \delta\boldsymbol{\tau} =$$

$$\delta\mathbf{F}_n \cdot \boldsymbol{\tau}(t_n) \cdot \mathbf{F}_n^c + \mathbf{F}_n \cdot \boldsymbol{\tau}(t_n) \cdot \delta\mathbf{F}_n^c + \delta {}^4\mathbf{H} : (\mathbf{e}_n - \Delta\lambda \mathbf{a}) + {}^4\mathbf{H} : \delta\mathbf{e}_n$$

$${}^4\mathbf{V} : \delta\boldsymbol{\tau} = {}^4\mathbf{E} : \delta\mathbf{F}_n \quad \rightarrow \quad \delta\boldsymbol{\tau} = {}^4\mathbf{V}^{-1} : {}^4\mathbf{E} : \delta\mathbf{F}_n$$

# Stiffness

$$\delta \mathbf{F}_n \cdot \boldsymbol{\tau}(t_n) \cdot \mathbf{F}_n^c + \mathbf{F}_n \cdot \boldsymbol{\tau}(t_n) \cdot \delta \mathbf{F}_n^c = {}^4\mathbf{T} : \delta \mathbf{F}_n$$

$$J = \det(\mathbf{F}_n) = \det(\mathbf{F}_n + \delta \mathbf{F}_n) = J(1 + \mathbf{F}_n^{-1} : \delta \mathbf{F}_n) \quad \rightarrow \quad \delta J = J \mathbf{F}_n^{-1} : \delta \mathbf{F}_n$$

$$\delta {}^4\mathbf{H} = \left( \frac{\partial {}^4\mathbf{H}}{\partial J} \right) \delta J = \left( \frac{\partial {}^4\mathbf{H}}{\partial J} \right) (J \mathbf{F}_n^{-1} : \delta \mathbf{F}_n)$$

$$\left. \begin{aligned} \delta \mathbf{e}_n &= -\frac{1}{2} \delta \mathbf{F}_n^{-c} \cdot \mathbf{F}_n^{-1} - \frac{1}{2} \mathbf{F}_n^{-c} \cdot \delta \mathbf{F}_n^{-1} = -{}^4\mathbf{A}_1 : \delta \mathbf{F}_n^{-1} \\ \delta \mathbf{F}_n^{-1} &= -\mathbf{F}_n^{-1} \cdot \delta \mathbf{F}_n \cdot \mathbf{F}_n^{-1} = -{}^4\mathbf{A}_2 : \delta \mathbf{F}_n \end{aligned} \right\} \rightarrow$$

$$\delta \mathbf{e}_n = ({}^4\mathbf{A}_1 : {}^4\mathbf{A}_2) : \delta \mathbf{F}_n = {}^4\mathbf{P} : \delta \mathbf{F}_n$$

# Consistent material stiffness tensor

$$\boldsymbol{\tau} = J\boldsymbol{\sigma} \quad \rightarrow \quad \boldsymbol{\sigma} = \frac{1}{J}\boldsymbol{\tau} \quad \rightarrow$$

$$\begin{aligned}\delta\boldsymbol{\sigma} &= \frac{1}{J}(\delta\boldsymbol{\tau} - \boldsymbol{\sigma}\delta J) \\ &= \frac{1}{J}\left\{{}^4\mathbf{V}^{-1} : {}^4\mathbf{E} - \boldsymbol{\sigma}J\mathbf{F}_n^{-1}\right\} : \delta\mathbf{F}_n \\ &= {}^4\mathbf{C} : \delta\mathbf{F}_n \\ &= {}^4\mathbf{M} : \mathbf{L}_u\end{aligned}$$

## Matrix/column notation

$$\begin{aligned}
 \delta \sigma &= {}^4\mathbf{C} : \delta \mathbf{F}_n & \rightarrow & \delta \underline{\underline{\sigma}} = \underline{\underline{C}} \delta \underline{\underline{F}}_n \\
 \delta \mathbf{F}_n &= (\mathbf{F}^{-c}(t_n) \cdot \delta \mathbf{F}^c)^c & \rightarrow & \delta \underline{\underline{F}}_n = \left( \underline{\underline{F}}_t^{-1}(t_n) \delta \underline{\underline{F}}_t \right)_t \rightarrow \delta \underline{\underline{F}}_n = \underline{\underline{F}}_t^{-1}(t_n) \delta \underline{\underline{F}}_t \\
 \delta \mathbf{F}^c &= \mathbf{F}^c \cdot \mathbf{L}_u^c & \rightarrow & \delta \underline{\underline{F}}_t = \underline{\underline{F}}_t \underline{\underline{L}}_u
 \end{aligned}$$

$$\delta \underline{\underline{\sigma}} = \left[ \underline{\underline{C}} \underline{\underline{F}}_t^{-1}(t_n) \underline{\underline{F}}_t \right] \underline{\underline{L}}_u = \underline{\underline{M}} \underline{\underline{L}}_u$$

$$\underline{\underline{M}} = \underline{\underline{C}} \underline{\underline{F}}_t^{-1}(t_n) \underline{\underline{F}}_t$$

$$\underline{\underline{C}} = \frac{1}{J} \left( \underline{\underline{V}}^{-1} \underline{\underline{E}}_r - J \underline{\underline{\sigma}} \underline{\underline{F}}_n^{-T} \right)$$

$$\underline{\underline{V}} = \underline{\underline{I}} + \Delta \lambda \underline{\underline{H}}_c \underline{\underline{b}} + c_1 \underline{\underline{H}}_c \underline{\underline{a}} \underline{\underline{a}}^T$$

$$\underline{\underline{E}} = \underline{\underline{I}} - 2\lambda \underline{\underline{I}} \left( \underline{\underline{e}}_n - \Delta \lambda \underline{\underline{a}} \right) \left( \underline{\underline{F}}_n^{-1} \right)^T + \underline{\underline{H}}_c \underline{\underline{P}}$$

# Plane strain

$$\delta J = J_1 \delta \lambda + \mathbf{J}_2 : \delta \boldsymbol{\tau} = 0$$

$$\delta \boldsymbol{\tau}_{tr} = \mathbf{M}_1 \delta \lambda + {}^4\mathbf{M}_2 : \delta \boldsymbol{\tau} = \mathbf{0}$$

# Iterative stress update

$$\left. \begin{aligned} {}^4\mathbf{R} : \delta\boldsymbol{\tau} + \mathbf{t}\delta\lambda &= -\mathbf{s}_1 \\ \mathbf{u} : \delta\boldsymbol{\tau} + v\delta\lambda &= -s_2 \end{aligned} \right\}$$

$${}^4\mathbf{R} = {}^4\mathbf{I} + \Delta\lambda {}^4\mathbf{H} : {}^4\mathbf{b}$$

$$\mathbf{t} = {}^4\mathbf{H} : \mathbf{a}$$

$$\mathbf{u} = -\Delta t \gamma \left( \frac{\partial \phi}{\partial F} \right) \mathbf{a}$$

$$v = 1 - \Delta t \gamma \left( \frac{\partial \phi}{\partial F} \right) \left( \frac{\partial F}{\partial \bar{\varepsilon}_{vp}} \right)$$

$$\mathbf{s}_1 = \boldsymbol{\tau} - \boldsymbol{\tau}_{tr} + \Delta\lambda {}^4\mathbf{H} : \mathbf{a}$$

$$s_2 = \Delta\lambda - \Delta t \gamma \phi(F)$$

## Matrix/column notation

$$\begin{bmatrix} \underline{\underline{R}}_c & \underline{\underline{t}} \\ \underline{\underline{u}}_t^T & \underline{\underline{v}} \end{bmatrix} \begin{bmatrix} \underline{\underline{\delta\tau}} \\ \underline{\underline{\delta\lambda}} \end{bmatrix} = - \begin{bmatrix} \underline{\underline{s}}_1 \\ \underline{\underline{s}}_2 \end{bmatrix}$$

$$\underline{\underline{R}} = \underline{\underline{I}} + \Delta\lambda \underline{\underline{H}} \underline{\underline{b}}_t$$

$$\underline{\underline{t}} = \underline{\underline{H}} \underline{\underline{a}}_t$$

$$\underline{\underline{u}} = -\Delta t \gamma \left( \frac{\partial \phi}{\partial F} \right) \underline{\underline{a}}$$

$$\underline{\underline{v}} = 1 - \Delta t \gamma \left( \frac{\partial \phi}{\partial F} \right) \left( \frac{\partial F}{\partial \bar{\epsilon}_{vp}} \right)$$

$$\underline{\underline{s}}_1 = \underline{\underline{\tau}} - \underline{\underline{\tau}}_{tr} + \Delta\lambda \underline{\underline{H}} \underline{\underline{a}}_t$$

$$\underline{\underline{s}}_2 = \Delta\lambda - \Delta t \gamma \phi(F)$$

# Stiffness

$$\delta \sigma = {}^4\mathbf{C} : \delta \mathbf{F}_n = \frac{1}{J} \left\{ {}^4\mathbf{V}^{-1} : {}^4\mathbf{E} - \sigma J \mathbf{F}_n^{-1} \right\} : \delta \mathbf{F}_n$$

$${}^4\mathbf{V} = \left\{ {}^4\mathbf{I} + \Delta\lambda {}^4\mathbf{H} : {}^4\mathbf{b} + c_1 {}^4\mathbf{H} : \mathbf{a}\mathbf{a} \right\}$$

$${}^4\mathbf{E} = \left\{ {}^4\mathbf{T} + {}^4\mathbf{c} : (\mathbf{e}_n - \Delta\lambda \mathbf{a}) J \mathbf{F}_n^{-1} + {}^4\mathbf{H} : {}^4\mathbf{P} \right\}$$

$$\delta \mathbf{F}_n \cdot \boldsymbol{\tau}(t_n) \cdot \mathbf{F}_n^c + \mathbf{F}_n \cdot \boldsymbol{\tau}(t_n) \cdot \delta \mathbf{F}_n^c = {}^4\mathbf{T} : \delta \mathbf{F}_n$$

$$\delta \mathbf{e}_n = {}^4\mathbf{P} : \delta \mathbf{F}_n$$



## Matrix/column notation

$$\delta \underline{\underline{g}} = \underline{\underline{C}} \left( \delta \underline{\underline{F}}_n \right)_t = \left[ \frac{1}{J} \left\{ \underline{\underline{V}}^{-1} \underline{\underline{E}}_r - \underline{\underline{g}} J \underline{\underline{F}}_n^{-T} \right\} \right] \left( \delta \underline{\underline{F}}_n \right)_t$$

$$\underline{\underline{V}} = \underline{\underline{I}} + \Delta \lambda \underline{\underline{H}}_c \underline{\underline{b}} + c_1 \underline{\underline{H}}_c \underline{\underline{a}} \underline{\underline{a}}^T$$

$$\underline{\underline{E}} = \underline{\underline{T}} + 2 \lambda \underline{\underline{I}} \left( \underline{\underline{e}} - \Delta \lambda \underline{\underline{a}} \right) J \underline{\underline{F}}_n^{-T} + \underline{\underline{H}}_c \underline{\underline{P}}$$

# Plane stress

## Iterative stress update

$$\left. \begin{aligned} {}^4\mathbf{R} : \delta\boldsymbol{\tau} + \mathbf{t}\delta\lambda &= -\mathbf{s}_1 \\ \mathbf{u} : \delta\boldsymbol{\tau} + v\delta\lambda &= -s_2 \end{aligned} \right\}$$

$${}^4\mathbf{R} = {}^4\mathbf{I} - {}^4\mathbf{M}_2 + \Delta\lambda {}^4\mathbf{C} : \mathbf{a}\mathbf{J}_2 + \Delta\lambda {}^4\mathbf{H} : {}^4\mathbf{b}$$

$$\mathbf{t} = -\mathbf{M}_1 + \Delta\lambda {}^4\mathbf{C} : \mathbf{a}\mathbf{J}_1 + {}^4\mathbf{H} : \mathbf{a}$$

$$\mathbf{u} = -\Delta t \gamma \left( \frac{\partial \phi}{\partial F} \right) \mathbf{a}$$

$$v = 1 - \Delta t \gamma \left( \frac{\partial \phi}{\partial F} \right) \left( \frac{\partial F}{\partial \bar{\varepsilon}_{vp}} \right)$$

$$\mathbf{s}_1 = \boldsymbol{\tau} - \boldsymbol{\tau}_{trial} + \Delta\lambda {}^4\mathbf{H} : \mathbf{a}$$

$$s_2 = \Delta\lambda - \Delta t \gamma \phi(F)$$

## Matrix/column notation

$$\begin{bmatrix} \underline{\underline{R}}_c & \underline{\underline{t}} \\ \underline{\underline{u}}_t^T & v \end{bmatrix} \begin{bmatrix} \delta \underline{\underline{\tau}} \\ \delta \lambda \end{bmatrix} = - \begin{bmatrix} \underline{\underline{s}}_1 \\ s_2 \end{bmatrix}$$

$$\underline{\underline{R}} = \underline{\underline{I}} - \underline{\underline{M}}_2 + \Delta \lambda \underline{\underline{C}}_{\approx r \approx 2} J_2^T + \Delta \lambda \underline{\underline{H}} \underline{\underline{b}}_r$$

$$\underline{\underline{t}} = -\underline{\underline{M}}_1 + \Delta \lambda \underline{\underline{C}}_{\approx t} J_1 + \underline{\underline{H}} \underline{\underline{a}}_t$$

$$\underline{\underline{u}} = -\Delta t \gamma \left( \frac{\partial \phi}{\partial F} \right)_{\approx} \underline{\underline{a}}$$

$$v = 1 - \Delta t \gamma \left( \frac{\partial \phi}{\partial F} \right) \left( \frac{\partial F}{\partial \bar{\varepsilon}_{vp}} \right)$$

$$\underline{\underline{s}}_1 = \underline{\underline{\tau}} - \underline{\underline{\tau}}_{tr} + \Delta \lambda \underline{\underline{H}} \underline{\underline{a}}_t$$

$$s_2 = \Delta \lambda - \Delta t \gamma \phi(F)$$

# Stiffness

$$\delta \boldsymbol{\sigma} = {}^4\mathbf{C} : \delta \mathbf{F}_n = \frac{1}{J} \left\{ {}^4\mathbf{V}^{-1} : {}^4\mathbf{E} - \sigma J \mathbf{F}_n^{-1} \right\} : \delta \mathbf{F}_n$$

$${}^4\mathbf{V} = \left\{ {}^4\mathbf{I} + \Delta\lambda {}^4\mathbf{H} : \left( \frac{\partial \mathbf{a}}{\partial \boldsymbol{\tau}} \right) + c_1 {}^4\mathbf{H} : \mathbf{a}\mathbf{a} \right\}$$

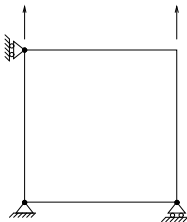
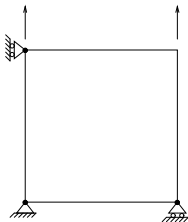
$${}^4\mathbf{E} = \left\{ {}^4\mathbf{T} + \left( \frac{\partial {}^4\mathbf{H}}{\partial J} \right) : (\mathbf{e} - \Delta\lambda \mathbf{a}) J \mathbf{F}_n^{-1} + {}^4\mathbf{H} : {}^4\mathbf{P} \right\}$$

$$\delta \mathbf{F}_n \cdot \boldsymbol{\tau}(t_n) \cdot \mathbf{F}_n^c + \mathbf{F}_n \cdot \boldsymbol{\tau}(t_n) \cdot \delta \mathbf{F}_n^c = {}^4\mathbf{T} : \delta \mathbf{F}_n$$



# Examples

# Tensile test



initial width	$w_0$	100	mm
initial height	$h_0$	100	mm
initial thickness	$d_0$	0.1	mm

initial radius	$r_0$	$\sqrt{(10/\pi)}$	mm
initial height	$h_0$	100	mm

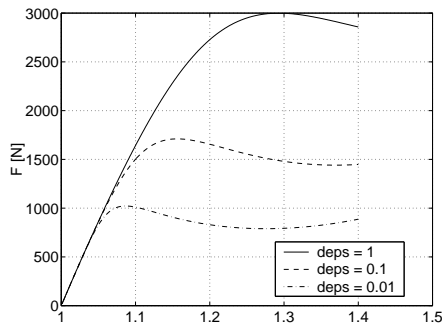
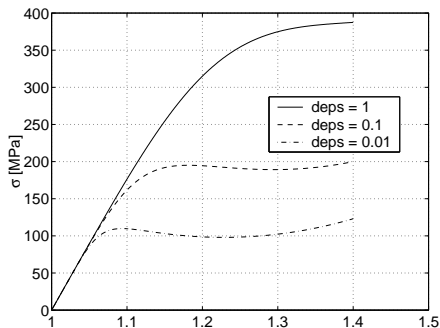


# Tensile test at various strain rates

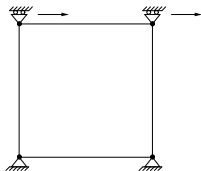
$E$	1800	MPa	$\nu$	0.37	-
$\sigma_{y0}$	37	MPa	$H$	-200	MPa
$\gamma$	0.001	1/s	$N$	3	-
$a$	500	MPa	$b$	700	MPa
$c$	800	MPa	$d$	30000	MPa

elongation rate

$$\frac{\dot{\Delta}l}{h_0} = \{0.01, 0.1, 1\} \text{ s}^{-1}$$



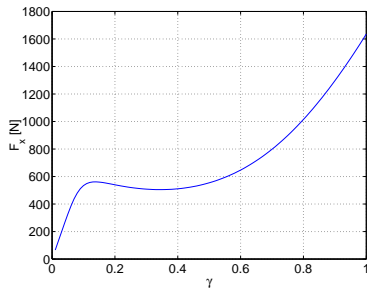
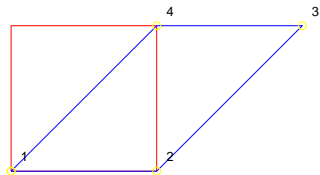
# Shear test



initial width	$w_0$	100	mm
initial height	$h_0$	100	mm
initial thickness	$d_0$	0.1	mm

strain rate

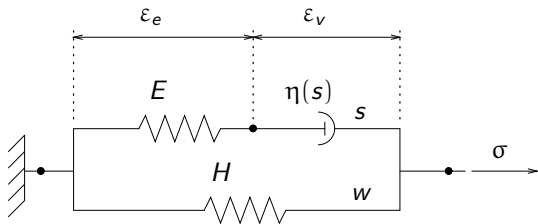
$$\dot{\gamma} = \frac{\dot{u}}{h_0} = 0.01 \text{ s}^{-1}$$



# NONLINEAR VISCOELASTIC

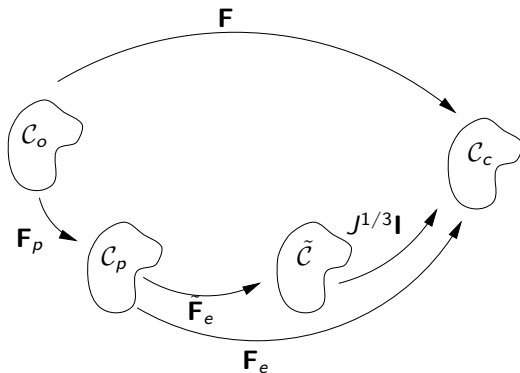
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# Nonlinear viscoelastic material behavior



$$\sigma = s + w$$

# Kinematics

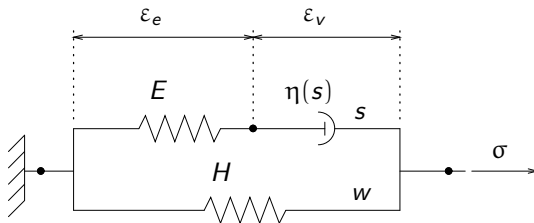


$$\mathbf{F} = (\vec{\nabla}_0 \vec{x})^c = \mathbf{F}_e \cdot \mathbf{F}_p = J^{1/3} \mathbf{I} \cdot \tilde{\mathbf{F}}_e \cdot \mathbf{F}_p$$

$$\mathbf{C} = \mathbf{F}^c \cdot \mathbf{F} \quad ; \quad \mathbf{B} = \mathbf{F} \cdot \mathbf{F}^c \rightarrow \tilde{\mathbf{B}}_e = \tilde{\mathbf{F}}_e \cdot \tilde{\mathbf{F}}_e^c$$

$$\begin{aligned} \mathbf{L} &= \dot{\mathbf{F}} \cdot \mathbf{F}^{-1} = (\vec{\nabla} \vec{v})^c \\ &= \mathbf{L}_e + \mathbf{L}_p = (\mathbf{D}_e + \boldsymbol{\Omega}_e) + (\mathbf{D}_p + \boldsymbol{\Omega}_p) = (\mathbf{D}_e + \boldsymbol{\Omega}_e) + \mathbf{D}_p \end{aligned}$$

# Stress decomposition



$$\sigma = s + w = s^d + s^h + w$$

$$s = G\tilde{\mathbf{B}}_e^d + \kappa(J-1)\mathbf{I} \quad ; \quad w = H\tilde{\mathbf{B}}^d$$

# Elastic deformation

$$\tilde{\mathbf{B}}_e = \tilde{\mathbf{F}}_e \cdot \tilde{\mathbf{F}}_e^c \rightarrow \dot{\tilde{\mathbf{B}}}_e = \dot{\tilde{\mathbf{F}}}_e \cdot \tilde{\mathbf{F}}_e^c + \tilde{\mathbf{F}}_e \cdot \dot{\tilde{\mathbf{F}}}_e^c$$

$$\tilde{\mathbf{F}} = \tilde{\mathbf{F}}_e \cdot \mathbf{F}_p \rightarrow \tilde{\mathbf{F}}_e = \tilde{\mathbf{F}} \cdot \mathbf{F}_p^{-1} \rightarrow \dot{\tilde{\mathbf{F}}}_e = \dot{\tilde{\mathbf{F}}} \cdot \mathbf{F}_p^{-1} + \tilde{\mathbf{F}} \cdot \dot{\mathbf{F}}_p^{-1}$$

$$\begin{aligned} \dot{\tilde{\mathbf{B}}}_e &= \left( \dot{\tilde{\mathbf{F}}} \cdot \mathbf{F}_p^{-1} + \tilde{\mathbf{F}} \cdot \dot{\mathbf{F}}_p^{-1} \right) \cdot \tilde{\mathbf{F}}_e^c + \tilde{\mathbf{F}}_e \cdot \left( \mathbf{F}_p^{-c} \cdot \dot{\tilde{\mathbf{F}}}^c + \dot{\mathbf{F}}_p^{-c} \cdot \tilde{\mathbf{F}}^c \right) \\ &= \left( \dot{\tilde{\mathbf{F}}} \cdot \mathbf{F}_p^{-1} \cdot \tilde{\mathbf{F}}_e^{-1} + \tilde{\mathbf{F}} \cdot \dot{\mathbf{F}}_p^{-1} \cdot \tilde{\mathbf{F}}_e^{-1} \right) \cdot \tilde{\mathbf{B}}_e + \\ &\quad \tilde{\mathbf{B}}_e \cdot \left( \tilde{\mathbf{F}}_e^{-c} \cdot \mathbf{F}_p^{-c} \cdot \dot{\tilde{\mathbf{F}}}^c + \tilde{\mathbf{F}}_e^{-c} \cdot \dot{\mathbf{F}}_p^{-c} \cdot \tilde{\mathbf{F}}^c \right) \\ &= \left( \tilde{\mathbf{L}} + \tilde{\mathbf{F}}_e \cdot \mathbf{F}_p \cdot \dot{\mathbf{F}}_p^{-1} \tilde{\mathbf{F}}_e^{-1} \right) \cdot \tilde{\mathbf{B}}_e + \tilde{\mathbf{B}}_e \cdot \left( \tilde{\mathbf{L}}^c + \tilde{\mathbf{F}}_e^{-c} \cdot \dot{\mathbf{F}}_p^{-c} \cdot \mathbf{F}_p^c \cdot \tilde{\mathbf{F}}_e^c \right) \\ &\quad \mathbf{F}_p \cdot \mathbf{F}_p^{-1} = \mathbf{I} \rightarrow \mathbf{F}_p \cdot \dot{\mathbf{F}}_p^{-1} = -\dot{\mathbf{F}}_p \cdot \mathbf{F}_p^{-1} \rightarrow \\ &= (\tilde{\mathbf{L}} - \mathbf{D}_p) \cdot \tilde{\mathbf{B}}_e + \tilde{\mathbf{B}}_e \cdot (\tilde{\mathbf{L}}^c - \mathbf{D}_p) \end{aligned}$$

# Viscoplastic deformation

$$\mathbf{D}_p = \frac{1}{2\eta} \mathbf{s}^d$$

$$\eta = \eta(\bar{s}, p, T, D)$$

$$\bar{s} = \sqrt{\frac{3}{2} \mathbf{s}^d : \mathbf{s}^d}$$

$$p = \kappa(J - 1)\mathbf{I}$$



# Eyring viscosity

plastic deformation rate

$$\mathbf{D}_p = \frac{1}{2\eta(\bar{s}, p, T, D)} \mathbf{s}^d$$

$$\eta = \frac{A\bar{s}}{\sqrt{3} \sinh\left(\frac{\bar{s}}{\sqrt{3}\tau_0}\right)}$$

$$\bar{s} = \sqrt{\frac{3}{2} \mathbf{s}^d : \mathbf{s}^d}$$

$$A = A_0 \exp\left[\frac{\Delta H}{RT} + \frac{\mu p}{\tau_0} - D\right]$$

$$\tau_0 = \frac{RT}{V} \quad ; \quad p = -\frac{1}{3} \text{tr}(\boldsymbol{\sigma})$$

$$\dot{D} = h \left(1 - \frac{D}{D_\infty}\right) \frac{\bar{s}}{\sqrt{6}\eta} \quad ; \quad D \in [0, D_\infty]$$

# Bodner-Partom viscosity

plastic deformation rate

$$\mathbf{D}_p = \frac{1}{2\eta(\bar{s}, \mathbf{D}_p)} \mathbf{s}^d$$

$$\eta = \frac{\bar{s}}{\sqrt{12}\Gamma_0} \exp \left[ \frac{1}{2} \left( \frac{Z}{\bar{\sigma}} \right)^{2n} \right]$$

$$\bar{s} = \sqrt{\frac{3}{2} \mathbf{s}^d : \mathbf{s}^d}$$

$$Z = Z_1 + (Z_0 - Z_1) e^{-m\bar{\epsilon}_p}$$

$$\dot{\bar{\epsilon}}_p = \sqrt{\frac{2}{3} \mathbf{D}_p : \mathbf{D}_p} \quad \rightarrow \quad \bar{\epsilon}_p$$

## Plastic strain rate

$$\left. \begin{aligned} \tilde{\mathbf{F}} &= \tilde{\mathbf{F}}_e \cdot \mathbf{F}_p \rightarrow \mathbf{C}_p = \mathbf{F}_p^c \cdot \mathbf{F}_p = \tilde{\mathbf{F}}^c \cdot \tilde{\mathbf{B}}_e^{-1} \cdot \tilde{\mathbf{F}} \rightarrow \\ \dot{\mathbf{C}}_p &= \tilde{\mathbf{F}}^c \cdot \tilde{\mathbf{B}}_e^{-1} \cdot \left[ \tilde{\mathbf{B}}_e \cdot \tilde{\mathbf{L}}^c + \tilde{\mathbf{B}}_e \cdot \dot{\tilde{\mathbf{B}}}_e^{-1} \cdot \tilde{\mathbf{B}}_e + \tilde{\mathbf{L}} \cdot \tilde{\mathbf{B}}_e \right] \cdot \tilde{\mathbf{B}}_e^{-1} \cdot \tilde{\mathbf{F}} \end{aligned} \right\} \rightarrow$$

$$\left. \begin{aligned} \dot{\tilde{\mathbf{B}}}_e &= (\tilde{\mathbf{L}} - \mathbf{D}_p) \cdot \tilde{\mathbf{B}}_e + \tilde{\mathbf{B}}_e \cdot (\tilde{\mathbf{L}}^c - \mathbf{D}_p) \rightarrow \\ \tilde{\mathbf{B}}_e \cdot \dot{\tilde{\mathbf{B}}}_e^{-1} &= -\tilde{\mathbf{L}} - \tilde{\mathbf{B}}_e \cdot \tilde{\mathbf{L}}^c \cdot \tilde{\mathbf{B}}_e^{-1} + \mathbf{D}_p + \tilde{\mathbf{B}}_e \cdot \mathbf{D}_p \cdot \tilde{\mathbf{B}}_e^{-1} \end{aligned} \right\}$$

$$\begin{aligned} \dot{\mathbf{C}}_p &= \tilde{\mathbf{F}}^c \cdot \tilde{\mathbf{B}}_e^{-1} \cdot \left[ \mathbf{D}_p \cdot \tilde{\mathbf{B}}_e + \tilde{\mathbf{B}}_e \cdot \mathbf{D}_p \right] \cdot \tilde{\mathbf{B}}_e^{-1} \cdot \tilde{\mathbf{F}} \\ &\quad \text{with } \mathbf{D}_p = \frac{1}{2\eta} \mathbf{s}^d = \frac{G}{2\eta} \tilde{\mathbf{B}}_e^d \rightarrow \\ &= \frac{G}{\eta} \left( \tilde{\mathbf{C}} - \frac{1}{3} \text{tr}(\tilde{\mathbf{B}}_e) \mathbf{C}_p \right) = \Gamma \left( \tilde{\mathbf{C}} - \frac{1}{\alpha} \mathbf{C}_p \right) = \Gamma \mathbf{A} \end{aligned}$$

# Constitutive model

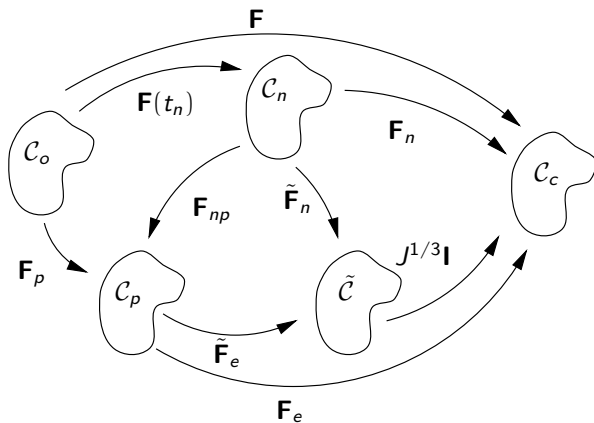
$$J = \det(\mathbf{F}) \quad \rightarrow \quad \tilde{\mathbf{F}} = J^{-1/3} \mathbf{F} \quad \rightarrow \quad \tilde{\mathbf{B}} = \tilde{\mathbf{F}} \cdot \tilde{\mathbf{F}}^c \quad \rightarrow \quad \mathbf{w} = H \tilde{\mathbf{B}}^d$$

$$p = \kappa(J - 1) \quad \rightarrow \quad \mathbf{s}^h = p \mathbf{I}$$

$$\left. \begin{aligned} \dot{\mathbf{C}}_p &= \frac{G}{\eta} \left( \tilde{\mathbf{C}} - \frac{1}{3} \text{tr}(\tilde{\mathbf{B}}_e) \mathbf{C}_p \right) \\ \tilde{\mathbf{B}}_e &= \tilde{\mathbf{F}} \cdot \mathbf{C}_p^{-1} \cdot \tilde{\mathbf{F}}^c \end{aligned} \right\} \quad \rightarrow \quad \mathbf{s}^d = G \tilde{\mathbf{B}}_e^d \rightarrow \bar{s} = \sqrt{\frac{3}{2} \mathbf{s}^d : \mathbf{s}^d}$$

$$\boldsymbol{\sigma} = \mathbf{s}^d + \mathbf{s}^h + \mathbf{w}$$

# Incremental analysis



$$\mathbf{F}(\tau) = \mathbf{F}_n(\tau) \cdot \mathbf{F}(t_n) \quad \rightarrow \quad \mathbf{F}_n(\tau) = \mathbf{F}(\tau) \cdot \mathbf{F}^{-1}(t_n)$$

$$\tilde{\mathbf{F}}(\tau) = \tilde{\mathbf{F}}_n(\tau) \cdot \tilde{\mathbf{F}}(t_n)$$

$$\mathbf{F}_n = \left( \vec{\nabla}_n \vec{X} \right)^c = \mathbf{R}_n \cdot \mathbf{U}_n$$

# Incremental plastic strain

$$\begin{aligned}\mathbf{C}_p(\tau) &= \mathbf{F}_p^c(\tau) \cdot \mathbf{F}_p(\tau) = \tilde{\mathbf{F}}^c(\tau) \cdot \tilde{\mathbf{B}}_e^{-1}(\tau) \cdot \tilde{\mathbf{F}}(\tau) \\ &\quad \text{with } \tilde{\mathbf{F}}(\tau) = \tilde{\mathbf{F}}_n(\tau) \cdot \tilde{\mathbf{F}}(t_n) \rightarrow \\ &= \tilde{\mathbf{F}}^c(t_n) \cdot \left[ \tilde{\mathbf{F}}_n^c(\tau) \cdot \tilde{\mathbf{B}}_e^{-1}(\tau) \cdot \tilde{\mathbf{F}}_n(\tau) \right] \cdot \tilde{\mathbf{F}}(t_n) \\ &= \tilde{\mathbf{F}}^c(t_n) \cdot \mathbf{C}_{p_n}(\tau) \cdot \tilde{\mathbf{F}}(t_n)\end{aligned}$$

incremental rotation neutralized plastic strain

$$\begin{aligned}\mathbf{C}_{p_n}(\tau) &= \tilde{\mathbf{F}}_n^c(\tau) \cdot \tilde{\mathbf{B}}_e^{-1}(\tau) \cdot \tilde{\mathbf{F}}_n(\tau) \\ &= \tilde{\mathbf{U}}_n(\tau) \cdot \left[ \mathbf{R}_n^c(\tau) \cdot \tilde{\mathbf{B}}_e^{-1}(\tau) \cdot \mathbf{R}_n(\tau) \right] \cdot \tilde{\mathbf{U}}_n(\tau) \\ &= \tilde{\mathbf{U}}_n(\tau) \cdot \tilde{\mathbf{B}}_{e_n}^{-1}(\tau) \cdot \tilde{\mathbf{U}}_n(\tau)\end{aligned}$$

# Constitutive equations

$$J = \det(\mathbf{F}) \quad \rightarrow \quad \tilde{\mathbf{F}} = J^{-1/3} \mathbf{F} \quad \rightarrow \quad \tilde{\mathbf{B}} = \tilde{\mathbf{F}} \cdot \tilde{\mathbf{F}}^c \quad \rightarrow \quad \mathbf{w} = H \tilde{\mathbf{B}}^d$$

$$p = \kappa(J - 1) \quad \rightarrow \quad \mathbf{s}^h = p \mathbf{I}$$

$$\left. \begin{aligned} \dot{\mathbf{C}}_{p_n} &= \frac{G}{\eta} \left( \tilde{\mathbf{C}}_n - \frac{1}{3} \text{tr}(\tilde{\mathbf{B}}_{e_n}) \mathbf{C}_{p_n} \right) \\ \tilde{\mathbf{B}}_{e_n} &= \tilde{\mathbf{U}}_n \cdot \mathbf{C}_{p_n}^{-1} \cdot \tilde{\mathbf{U}}_n^c \rightarrow \tilde{\mathbf{B}}_e = \mathbf{R}_n \cdot \tilde{\mathbf{B}}_{e_n} \cdot \mathbf{R}_n^c \\ \dot{D} &= h \left( 1 - \frac{D}{D_\infty} \right) \frac{\bar{s}}{\sqrt{6} \eta} \\ \eta &= \eta(\bar{s}, p, T, D) \end{aligned} \right\} \rightarrow \mathbf{s}^d = G \tilde{\mathbf{B}}_e^d \rightarrow \bar{s} = \sqrt{\frac{3}{2} \mathbf{s}^d : \mathbf{s}^d}$$

$$\boldsymbol{\sigma} = \mathbf{s}^d + \mathbf{s}^h + \mathbf{w}$$

## Stress update

$$\dot{\mathbf{C}}_{p_n}(\tau) = \Gamma(\tau) \left[ \tilde{\mathbf{C}}_n(\tau) - \frac{1}{\bar{\alpha}_n(\tau)} \mathbf{C}_{p_n}(\tau) \right] \quad ; \quad \frac{1}{\bar{\alpha}_n} = \frac{1}{3} \text{tr} \left( \tilde{\tilde{\mathbf{B}}}_{e_n} \right)$$

$$\frac{1}{\Delta t} [\mathbf{C}_{p_n} - \mathbf{C}_{p_n}(t_n)] = \Gamma \left[ \tilde{\mathbf{C}}_n - \frac{1}{\bar{\alpha}_n} \mathbf{C}_{p_n} \right] \rightarrow$$

$$\mathbf{C}_{p_n} = \frac{\bar{\alpha}_n \Delta t \Gamma}{\bar{\alpha}_n + \Delta t \Gamma} \tilde{\mathbf{C}}_n + \frac{\bar{\alpha}_n}{\bar{\alpha}_n + \Delta t \Gamma} \mathbf{C}_{p_n}(t_n) \rightarrow$$

$$\mathbf{C}_{p_n} = \bar{\alpha}_n(1 - \lambda) \tilde{\mathbf{C}}_n + \lambda \mathbf{C}_{p_n}(t_n) \quad ; \quad \lambda = \frac{\bar{\alpha}_n}{\bar{\alpha}_n + \Delta t \Gamma} = \text{elasticity parameter}$$

$$\tilde{\mathbf{B}}_e = \mathbf{R}_n \cdot \tilde{\tilde{\mathbf{B}}}_{e_n} \cdot \mathbf{R}_n^c = \tilde{\mathbf{F}}_n \cdot \mathbf{C}_{p_n}^{-1} \cdot \tilde{\mathbf{F}}_n^c$$



# Sub-incremental plastic strain update

$$\dot{\mathbf{C}}_{p_n}(\tau) = \Gamma(\tau) \left[ \tilde{\mathbf{C}}_n(\tau) - \frac{1}{\bar{\alpha}_n(\tau)} \mathbf{C}_{p_n}(\tau) \right] \quad ; \quad \frac{1}{\bar{\alpha}_n} = \frac{1}{3} \text{tr} \left( \bar{\bar{\mathbf{B}}}_{e_n} \right)$$

$$\left. \begin{array}{l} \text{sub-incremental deformation :} \quad j = 1 \cdots ns + 1 \\ j = 1 \quad : \quad \tau = t_n \quad ; \quad j = ns + 1 \quad : \quad \tau = t_{n+1} \\ \delta t = \Delta t / ns \quad ; \quad \delta \tilde{\mathbf{C}}_n = \left\{ \tilde{\mathbf{C}}_n \right\}^{1/ns} \quad ; \quad \tilde{\mathbf{C}}_n^j = \left\{ \delta \tilde{\mathbf{C}}_n \right\}^j \end{array} \right\}$$

$$\frac{1}{\delta t} \left[ \mathbf{C}_{p_n}^j - \mathbf{C}_{p_n}^{j-1} \right] = \Gamma^j \left[ \tilde{\mathbf{C}}_n^j - \frac{1}{\bar{\alpha}_n^j} \mathbf{C}_{p_n}^j \right] \rightarrow$$

$$\mathbf{C}_{p_n}^j = \frac{\bar{\alpha}_n^j \delta t \Gamma^j}{\bar{\alpha}_n^j + \delta t \Gamma^j} \tilde{\mathbf{C}}_n^j + \frac{\bar{\alpha}_n^j}{\bar{\alpha}_n^j + \delta t \Gamma^j} \mathbf{C}_{p_n}^{j-1} \rightarrow$$

$$\mathbf{C}_{p_n}^j = \bar{\alpha}_n^j (1 - \lambda^j) \tilde{\mathbf{C}}_n^j + \lambda^j \mathbf{C}_{p_n}^{j-1} \quad ; \quad \lambda^j = \frac{\bar{\alpha}_n^j}{\bar{\alpha}_n^j + \delta t \Gamma^j}$$

incremental plastic strain

total isochoric elastic strain

$$\mathbf{C}_{p_n} = \mathbf{C}_{p_n}(t_{n+1}) = \mathbf{C}_{p_n}^{ns+1}$$

$$\tilde{\mathbf{B}}_e = \mathbf{R}_n \cdot \bar{\bar{\mathbf{B}}}_{e_n} \cdot \mathbf{R}_n^c = \tilde{\mathbf{F}}_n \cdot \mathbf{C}_{p_n}^{-1} \cdot \tilde{\mathbf{F}}_n^c$$

## Scalar variable update

$$\lambda = 1/(1 + \Delta t \Gamma) \quad \rightarrow \quad f(\lambda, D) = \lambda(1 + \Delta t \Gamma) = 1$$

$$\frac{1}{\Delta t} \{D - D(t_n)\} = \dot{D} \quad \rightarrow \quad g(\lambda, D) = D - \Delta t \dot{D} = D(t_n)$$

Newton-Raphson iterative solution procedure

$$\begin{bmatrix} \frac{\partial f}{\partial \lambda} & \frac{\partial f}{\partial D} \\ \frac{\partial g}{\partial \lambda} & \frac{\partial g}{\partial D} \end{bmatrix} \begin{bmatrix} \delta \lambda \\ \delta D \end{bmatrix} = \begin{bmatrix} 1 - f^* \\ D(t_n) - g^* \end{bmatrix} = \begin{bmatrix} r_\lambda^* \\ r_D^* \end{bmatrix}$$

## Partial derivatives

$$\begin{aligned}\frac{\partial f}{\partial \lambda} &= 1 + \Delta t \Gamma + \lambda \Delta t \frac{\partial \Gamma}{\partial \lambda} = 1 + \Delta t \Gamma - \lambda \Delta t \frac{G}{\eta^2} \frac{\partial \eta}{\partial \bar{\sigma}} \frac{\partial \bar{\sigma}}{\partial \lambda} \\ &= 1 + \Delta t \Gamma - \lambda \Delta t \frac{G}{\eta^2} \left[ \eta \left( \frac{1}{\bar{\sigma}} - \frac{1}{\sqrt{3}\tau_0} \right) \right] \bar{\sigma}\end{aligned}$$

$$\frac{\partial f}{\partial D} = \lambda \Delta t \frac{\partial \Gamma}{\partial D} = -\lambda \Delta t \frac{G}{\eta^2} \frac{\partial \eta}{\partial D} = \lambda \Delta t \frac{G}{\eta^2} \eta = \lambda \Delta t \Gamma$$

$$\frac{\partial g}{\partial \lambda} = -\Delta t \frac{\partial \dot{D}}{\partial \lambda} = -\Delta t \frac{\partial \dot{D}}{\partial \bar{\sigma}} \frac{\partial \bar{\sigma}}{\partial \lambda} = -\Delta t \left[ \frac{\dot{D}}{\sqrt{3}\tau_0} \right] \bar{\sigma}$$

$$\frac{\partial g}{\partial D} = 1 - \Delta t \frac{\partial \dot{D}}{\partial D} = 1 - \Delta t \left[ \dot{D} - \frac{h\bar{\sigma}}{\sqrt{6}D_\infty \eta} \right]$$

# Matrix/column notation

$$J = \det(\underline{F}) \rightarrow \underline{\tilde{F}} = J^{-1/3} \underline{F} \rightarrow \underline{\tilde{B}} = \underline{\tilde{F}} \underline{\tilde{F}}^T \rightarrow \underline{w} = H \underline{\tilde{B}}^d$$

$$p = \kappa(J - 1) \rightarrow \underline{s}^h = p \underline{l}$$

$$\left. \begin{aligned} \lambda &= 1/(1 + \Delta t \Gamma) \\ \frac{1}{\Delta t} \{D - D(t_n)\} &= \dot{D} \\ \underline{C}_{p_n} &= (1 - \lambda) \underline{\tilde{C}}_n + \lambda \underline{C}_{p_n}(t_n) \\ \underline{\tilde{B}}_{e_n} &= \underline{\tilde{U}}_n \underline{C}_{p_n}^{-1} \underline{\tilde{U}}_n^T \\ \underline{\bar{s}}^d &= G \underline{\tilde{B}}_{e_n} \rightarrow \bar{s} = \sqrt{\frac{3}{2} \text{tr}(\underline{\bar{s}}^d \underline{\bar{s}}^d)} \\ \eta &= \eta(\bar{s}, p, T, D) \end{aligned} \right\} \rightarrow \underline{\tilde{B}}_{e_n} \rightarrow \left\{ \begin{aligned} \underline{\tilde{B}}_e &= \underline{R}_n \underline{\tilde{B}}_{e_n} \underline{R}_n^T \\ \underline{s}^d &= G \underline{\tilde{B}}_e^d \end{aligned} \right. \rightarrow$$

$$\underline{\sigma} = \underline{s}^d + \underline{s}^h + \underline{w}$$

# Stiffness

$$\left. \begin{aligned} \boldsymbol{\sigma} &= \mathbf{s}^d + \mathbf{s}^h + \mathbf{w} = G \tilde{\mathbf{B}}_e^d + \kappa \mathbf{I}(J - 1) + H \tilde{\mathbf{B}}^d \\ \tilde{\mathbf{B}}_e &= \tilde{\mathbf{F}} \cdot \mathbf{C}_p^{-1} \cdot \tilde{\mathbf{F}}^c \\ \mathbf{C}_p &= (1 - \lambda) \tilde{\mathbf{C}} + \lambda \mathbf{C}_p(t_n) \\ \tilde{\mathbf{F}} &= J^{-1/3} \mathbf{F} \end{aligned} \right\}$$

$$\begin{aligned} \delta \boldsymbol{\sigma} &= \delta \mathbf{s}^d + \delta \mathbf{s}^h + \delta \mathbf{w} \\ &= G \delta \tilde{\mathbf{B}}_e^d + \kappa \mathbf{I} \delta J + H \delta \tilde{\mathbf{B}}^d = ({}^4\mathbf{S}_d + {}^4\mathbf{S}_h + {}^4\mathbf{H}) : \delta \mathbf{F} \\ &= {}^4\mathbf{S} : \delta \mathbf{F} = {}^4\mathbf{S}^{rc} : \delta \mathbf{F}^c \quad \text{with} \quad \delta \mathbf{F}^c = \vec{\nabla}_0 \vec{u} = \mathbf{F}^c \cdot \vec{\nabla} \vec{u} = \mathbf{F}^c \cdot \mathbf{L}_u^c \\ &= {}^4\mathbf{S}^{rc} : (\mathbf{F}^c \cdot \mathbf{L}_u^c) \\ &= {}^4\mathbf{M} : \mathbf{L}_u^c \end{aligned}$$

## Elastic strain variation

$$\tilde{\mathbf{B}}_e = \tilde{\mathbf{F}} \cdot \mathbf{C}_p^{-1} \cdot \tilde{\mathbf{F}}^c$$

$$\begin{aligned}\delta \tilde{\mathbf{B}}_e &= \delta \tilde{\mathbf{F}} \cdot \mathbf{C}_p^{-1} \cdot \tilde{\mathbf{F}}^c - \tilde{\mathbf{F}} \cdot \mathbf{C}_p^{-1} \cdot \delta \mathbf{C}_p \cdot \mathbf{C}_p^{-1} \cdot \tilde{\mathbf{F}}^c + \tilde{\mathbf{F}} \cdot \mathbf{C}_p^{-1} \cdot \delta \tilde{\mathbf{F}}^c \\ &= \left( \tilde{\mathbf{F}} \cdot \mathbf{C}_p^{-c} \cdot \delta \tilde{\mathbf{F}}^c \right)^c - \tilde{\mathbf{F}} \cdot \mathbf{C}_p^{-1} \cdot \left( \tilde{\mathbf{F}} \cdot \mathbf{C}_p^{-c} \cdot \delta \mathbf{C}_p^c \right)^c + \tilde{\mathbf{F}} \cdot \mathbf{C}_p^{-1} \cdot \delta \tilde{\mathbf{F}}^c \\ &= \left( \mathbf{M}^{(1)} \cdot \delta \tilde{\mathbf{F}}^c \right)^c - \mathbf{M}^{(2)} \cdot \left( \mathbf{M}^{(1)} \cdot \delta \mathbf{C}_p^c \right)^c + \mathbf{M}^{(2)} \cdot \delta \tilde{\mathbf{F}}^c\end{aligned}$$

$$\tilde{\mathbf{B}}_e^d = \tilde{\mathbf{B}}_e - \frac{1}{3} \text{tr}(\tilde{\mathbf{B}}_e) \mathbf{I} = ({}^4\mathbf{I} - \frac{1}{3}\mathbf{II}) : \tilde{\mathbf{B}}_e$$

$$\delta \tilde{\mathbf{B}}_e^d = ({}^4\mathbf{I} - \frac{1}{3}\mathbf{II}) : \delta \tilde{\mathbf{B}}_e$$

## Plastic strain variation

$$\begin{aligned}\mathbf{C}_p &= (1 - \lambda)\tilde{\mathbf{C}} + \lambda\mathbf{C}_p(t_n) \\ \delta\mathbf{C}_p &= (1 - \lambda)\delta\tilde{\mathbf{C}} + \left(\mathbf{C}_p(t_n) - \tilde{\mathbf{C}}\right)\delta\lambda \\ &= (1 - \lambda)\left(\delta\tilde{\mathbf{F}}^c \cdot \tilde{\mathbf{F}} + \tilde{\mathbf{F}}^c \cdot \delta\tilde{\mathbf{F}}\right) + \left(\mathbf{C}_p(t_n) - \tilde{\mathbf{C}}\right)\delta\lambda \\ &= (1 - \lambda)\left[\left(\tilde{\mathbf{F}}^c \cdot \delta\tilde{\mathbf{F}}\right)^c + \tilde{\mathbf{F}}^c \cdot \delta\tilde{\mathbf{F}}\right] + \left(\mathbf{C}_p(t_n) - \tilde{\mathbf{C}}\right)\delta\lambda\end{aligned}$$

# Deformation tensor variation

$$\begin{aligned}\tilde{\mathbf{F}} &= J^{-1/3} \mathbf{F} \\ \delta \tilde{\mathbf{F}} &= -\frac{1}{6} J^{-1/3} \mathbf{F} \mathbf{I} : (\delta \mathbf{F} \cdot \mathbf{F}^{-1} + \mathbf{F}^{-c} \cdot \delta \mathbf{F}^c) + J^{-1/3} \delta \mathbf{F} \\ &= -\frac{1}{3} J^{-1/3} \mathbf{F} (\mathbf{F}^{-c} : \delta \mathbf{F}^c) + J^{-1/3} \delta \mathbf{F}\end{aligned}$$



# Elasticity scalar variation

$$\delta\lambda = \frac{\lambda\Delta t\Gamma}{G\Delta t + \eta} \delta\eta = l_1 \tilde{\mathbf{B}}_e^d : \delta\tilde{\mathbf{B}}_e + l_2 \mathbf{I} : \delta\mathbf{F}$$

$$l_1 = \frac{\lambda\Delta t\Gamma h_1}{\Delta t G + \eta} \quad ; \quad l_2 = \frac{l_1 h_2}{h_1}$$

$$h_1 = \frac{3G^2}{2\bar{\sigma}} \left( \frac{\partial\eta}{\partial\bar{\sigma}} + \frac{\partial\eta}{\partial D} \frac{\partial D}{\partial\bar{\sigma}} \right) \quad ; \quad h_2 = -\kappa J \left( \frac{\partial\eta}{\partial p} + \frac{\partial\eta}{\partial D} \frac{\partial D}{\partial p} \right)$$

$$\frac{\partial\eta}{\partial\bar{\sigma}} = \eta \left( \frac{1}{\bar{\sigma}} - \frac{1}{\sqrt{3}\tau_0} \right) \quad ; \quad \frac{\partial\eta}{\partial p} = \frac{\eta\mu}{\tau_0} \quad ; \quad \frac{\partial\eta}{\partial D} = -\eta$$

$$\frac{\partial D}{\partial\bar{\sigma}} = \frac{\Delta t \frac{\partial\dot{D}}{\partial\bar{\sigma}}}{1 - \Delta t \frac{\partial\dot{D}}{\partial\bar{\sigma}}} \quad ; \quad \frac{\partial D}{\partial p} = \frac{\Delta t \frac{\partial\dot{D}}{\partial p}}{1 - \Delta t \frac{\partial\dot{D}}{\partial p}}$$

$$\frac{\partial\dot{D}}{\partial\bar{\sigma}} = \frac{\dot{D}}{\sqrt{3}\tau_0} \quad ; \quad \frac{\partial\dot{D}}{\partial p} = -\frac{\dot{D}\mu}{\tau_0} \quad ; \quad \frac{\partial\dot{D}}{\partial D} = \dot{D} - \frac{h\bar{\sigma}}{\sqrt{6}D_\infty\eta}$$

$$\text{with} \quad \dot{D} = h \left( 1 - \frac{D}{D_\infty} \right) \frac{\bar{\sigma}}{\sqrt{6}\eta}$$

## Deviatoric stress variation

$$\delta \mathbf{s}^d = G \delta \tilde{\mathbf{B}}_e^d = {}^4\mathbf{S}_d : \delta \mathbf{F}$$

# Hydrostatic stress variation

$$\delta \mathbf{s}^h = \kappa \mathbf{I} \delta J = {}^4\mathbf{S}_h : \delta \mathbf{F}$$

$$\dot{J} = J \operatorname{tr}(\mathbf{D}) = J \frac{1}{2} \operatorname{tr} \left\{ \dot{\mathbf{F}} \cdot \mathbf{F}^{-1} + \left( \dot{\mathbf{F}} \cdot \mathbf{F}^{-1} \right)^c \right\} \quad \rightarrow$$

$$\begin{aligned} \delta J &= \frac{1}{2} J \operatorname{tr} (\delta \mathbf{F} \cdot \mathbf{F}^{-1} + \mathbf{F}^{-c} \cdot \delta \mathbf{F}^c) \\ &= \frac{1}{2} J (\mathbf{F}^{-c} : \delta \mathbf{F}^c) + \frac{1}{2} J (\mathbf{F}^{-c} : \delta \mathbf{F}^c) \\ &= J \mathbf{F}^{-c} : \delta \mathbf{F}^c = J \mathbf{F}^{-1} : \delta \mathbf{F} \end{aligned}$$

## Hardening stress variation

$$\delta \mathbf{w} = H \delta \tilde{\mathbf{B}}^d = {}^4\mathbf{H} : \delta \mathbf{F}$$

$$\tilde{\mathbf{B}} = \tilde{\mathbf{F}} \cdot \tilde{\mathbf{F}}^c$$

$$\delta \tilde{\mathbf{B}} = \delta \tilde{\mathbf{F}} \cdot \tilde{\mathbf{F}}^c + \tilde{\mathbf{F}} \cdot \delta \tilde{\mathbf{F}}^c$$

$$\tilde{\mathbf{B}}^d = \tilde{\mathbf{B}} - \frac{1}{3} \text{tr}(\tilde{\mathbf{B}}) \mathbf{I} = ({}^4\mathbf{I} - \frac{1}{3} \mathbf{II}) : \tilde{\mathbf{B}}$$

$$\delta \tilde{\mathbf{B}}^d = ({}^4\mathbf{I} - \frac{1}{3} \mathbf{II}) : \left\{ \left( \tilde{\mathbf{F}} \cdot \delta \tilde{\mathbf{F}}^c \right)^c + \tilde{\mathbf{F}} \cdot \delta \tilde{\mathbf{F}}^c \right\}$$

# Consistent material stiffness tensor

$$\begin{aligned}\delta \boldsymbol{\sigma} &= \delta \mathbf{s}^d + \delta \mathbf{s}^h + \delta \mathbf{w} \\ &= ({}^4\mathbf{S}_d + {}^4\mathbf{S}_h + {}^4\mathbf{H}) : \delta \mathbf{F} = {}^4\mathbf{S} : \delta \mathbf{F} = {}^4\mathbf{S}^{rc} : \delta \mathbf{F}^c \\ &\quad \text{with } \delta \mathbf{F}^c = \vec{\nabla}_0 \vec{u} = \mathbf{F}^c \cdot \vec{\nabla} \vec{u} = \mathbf{F}^c \cdot \mathbf{L}_u^c \rightarrow \\ &= {}^4\mathbf{S}^{rc} : (\mathbf{F}^c \cdot \mathbf{L}_u^c) = {}^4\mathbf{M} : \mathbf{L}_u^c\end{aligned}$$

## Matrix/column notation

$$\delta \tilde{\underline{B}}_e = \left( \underline{M}_{cr}^{(1)} + \underline{M}_c^{(2)} \right) \delta \tilde{\underline{F}} - \underline{M}_c^{(2)} \underline{M}_c^{(1)} \delta \tilde{\underline{C}}_p \quad ; \quad \underline{M}^{(1)} = \tilde{\underline{F}} \underline{C}_p^{-T} ; \underline{M}^{(2)} = \tilde{\underline{F}} \underline{C}_p^{-1} \\ = \underline{A}^{(1)} \delta \tilde{\underline{F}} + \underline{A}^{(2)} \delta \tilde{\underline{C}}_p$$

$$\delta \tilde{\underline{B}}_e^d = \left( \underline{I} - \frac{1}{3} \underline{I} \underline{I}^T \right) \delta \tilde{\underline{B}}_e$$

$$\delta \tilde{\underline{C}}_p = \left[ (1 - \lambda) \left( \tilde{\underline{F}}_{tr} + \tilde{\underline{F}}_t \right) \right] \delta \tilde{\underline{F}} + \left( \tilde{\underline{C}}_p(t_n) - \tilde{\underline{C}} \right) \delta \lambda = \underline{\underline{C}}^{(1)} \delta \tilde{\underline{F}} + \underline{\underline{C}}^{(2)} \delta \lambda$$

$$\delta \tilde{\underline{F}} = \left[ -\frac{1}{3} J^{-1/3} \underline{F} \left( \underline{F}^{-1} \right)_t^T + J^{-1/3} \underline{I} \right] \delta \underline{F} = \underline{\underline{F}} \delta \underline{F}$$

$$\delta \lambda = = l_1 \left( \tilde{\underline{B}}_e^d \right)_t^T \delta \tilde{\underline{B}}_e + l_2 \underline{I}_t^T \delta \underline{F}$$

## Matrix/column notation

$$\begin{aligned}\delta \tilde{\underline{\underline{B}}}_e &= \underline{\underline{A}}^{(1)} \delta \tilde{\underline{\underline{F}}} + \underline{\underline{A}}^{(2)} \delta \tilde{\underline{\underline{C}}}_p \\ &= \left( \underline{\underline{A}}^{(1)} + \underline{\underline{A}}^{(2)} \underline{\underline{C}}^{(1)} \right) \delta \tilde{\underline{\underline{F}}} + \underline{\underline{A}}^{(2)} \underline{\underline{C}}^{(2)} \delta \lambda = \underline{\underline{B}}^{(1)} \delta \tilde{\underline{\underline{F}}} + \underline{\underline{B}}^{(2)} \delta \lambda\end{aligned}$$

$$= \underline{\underline{B}}^{(1)} \underline{\underline{F}} \delta \tilde{\underline{\underline{F}}} + l_1 \underline{\underline{B}}^{(2)} \left( \tilde{\underline{\underline{B}}}_e^d \right)_t^T \delta \tilde{\underline{\underline{B}}}_e + l_2 \underline{\underline{B}}^{(2)} \underline{\underline{I}}_t^T \delta \tilde{\underline{\underline{F}}}$$

$$\delta \tilde{\underline{\underline{B}}}_e = \left[ \underline{\underline{I}} - l_1 \underline{\underline{B}}^{(2)} \left( \tilde{\underline{\underline{B}}}_e^d \right)_t^T \right]^{-1} \left[ \underline{\underline{B}}^{(1)} \underline{\underline{F}} + l_2 \underline{\underline{B}}^{(2)} \underline{\underline{I}}_t^T \right] \delta \tilde{\underline{\underline{F}}}$$

$$\begin{aligned}\delta \tilde{\underline{\underline{B}}}_e^d &= \left( \underline{\underline{I}} - \frac{1}{3} \underline{\underline{I}} \underline{\underline{I}}_t^T \right) \delta \tilde{\underline{\underline{B}}}_e \\ &= \left( \underline{\underline{I}} - \frac{1}{3} \underline{\underline{I}} \underline{\underline{I}}_t^T \right) \left[ \underline{\underline{I}} - l_1 \underline{\underline{B}}^{(2)} \left( \tilde{\underline{\underline{B}}}_e^d \right)_t^T \right]^{-1} \left[ \underline{\underline{B}}^{(1)} \underline{\underline{F}} + l_2 \underline{\underline{B}}^{(2)} \underline{\underline{I}}_t^T \right] \delta \tilde{\underline{\underline{F}}} = \underline{\underline{B}}^{(3)} \delta \tilde{\underline{\underline{F}}}\end{aligned}$$

$$\begin{aligned}\delta \tilde{\underline{\underline{B}}}_e^d &= \left( \underline{\underline{I}} - \frac{1}{3} \underline{\underline{I}} \underline{\underline{I}}_t^T \right) \left( \tilde{\underline{\underline{F}}}_{cr} + \tilde{\underline{\underline{F}}}_c \right) \delta \tilde{\underline{\underline{F}}} \\ &= \left( \tilde{\underline{\underline{F}}}_{cr} + \tilde{\underline{\underline{F}}}_c - \frac{2}{3} \underline{\underline{I}} \underline{\underline{I}}_t^T \tilde{\underline{\underline{F}}}_c \right) \underline{\underline{F}} \delta \tilde{\underline{\underline{F}}} = \underline{\underline{B}}^{(4)} \delta \tilde{\underline{\underline{F}}}\end{aligned}$$

## Matrix/column notation

$$\delta \underline{\underline{s}}^d = G \delta \underline{\underline{B}}_e^d = G \underline{\underline{B}}^{(3)} \delta \underline{\underline{F}} = \underline{\underline{S}}_d \delta \underline{\underline{F}}$$

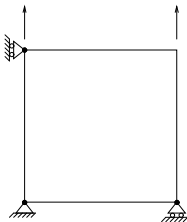
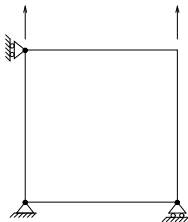
$$\delta \underline{\underline{s}}^h = \kappa \underline{\underline{I}} \delta J = \kappa \underline{\underline{J}} \underline{\underline{I}} (\underline{\underline{F}}^{-1})_t^T \delta \underline{\underline{F}} = \underline{\underline{S}}_h \delta \underline{\underline{F}}$$

$$\delta \underline{\underline{w}} = H \delta \underline{\underline{B}}^d = H \underline{\underline{B}}^{(4)} \delta \underline{\underline{F}} = \underline{\underline{H}} \delta \underline{\underline{F}}$$

$$\begin{aligned} \delta \underline{\underline{\sigma}} &= \delta \underline{\underline{s}}^d + \delta \underline{\underline{s}}^h + \delta \underline{\underline{w}} \\ &= (\underline{\underline{S}}_d + \underline{\underline{S}}_h + \underline{\underline{H}}) \delta \underline{\underline{F}} = \underline{\underline{S}} \delta \underline{\underline{F}} = \underline{\underline{S}}_c \delta \underline{\underline{F}}_t \\ &\quad \text{with } \delta \underline{\underline{F}}_t = \underline{\underline{F}}_t \left( \underline{\underline{L}}_u \right)_t \\ &= \underline{\underline{S}}_c \underline{\underline{F}}_t \left( \underline{\underline{L}}_u \right)_t \\ &= \underline{\underline{M}} \left( \underline{\underline{L}}_u \right)_t \end{aligned}$$



# Tensile test



initial width	$w_0$	100	mm
initial height	$h_0$	100	mm
initial thickness	$d_0$	0.1	mm

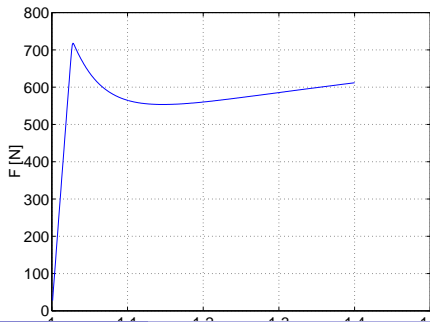
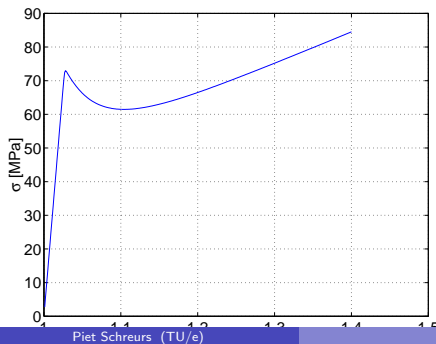
initial radius	$r_0$	$\sqrt{(10/\pi)}$	mm
initial height	$h_0$	100	mm

# Viscoelastic model in tensile test

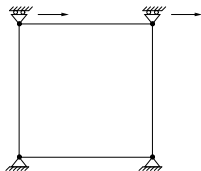
$E$	2305	MPa	$\nu$	0.37	-
$H$	29	MPa	$h$	270	-
$D_{\infty}$	19	-	$A_0$	9.7573E-27	s
$\Delta H$	2.9E5	J/mol	$\mu$	0.06984	-
$\tau_0$	0.72	MPa			

elongation rate

$$\frac{\dot{\Delta}l}{h_0} = 0.01 \text{ s}^{-1}$$



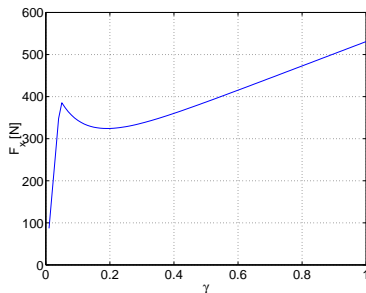
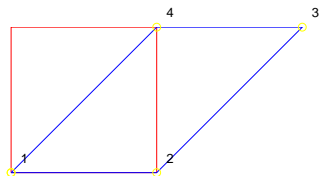
# Shear test





initial width	$w_0$	100	mm
initial height	$h_0$	100	mm
initial thickness	$d_0$	0.1	mm

strain rate

$$\dot{\gamma} = \frac{\dot{u}}{h_0} = 0.01 \text{ s}^{-1}$$



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# APPENDICES

[back to index](#)

## Utilities m2cc.m and m2mm.m

[back to index](#)

## m2cc

```
%*****  
function [C] = m2cc(m,s);  
  
C = zeros(s,1);  
  
if s==9  
    C = [m(1,1); m(2,2); m(3,3);  
         m(1,2); m(2,1); m(2,3); m(3,2); m(3,1); m(1,3)];  
elseif s==5  
    C = [m(1,1); m(2,2); m(3,3); m(1,2); m(2,1)];  
elseif s==4  
    C = [m(1,1); m(2,2); m(1,2); m(2,1)];  
end;  
%*****
```



## m2mm

```
%*****
function [M] = m2mm(m,s);

M = zeros(s);

if s==9
    M = [ m(1,1) 0      0      0      m(1,2) 0      0      m(1,3) 0
          0      m(2,2) 0      m(2,1) 0      0      m(2,3) 0      0
          0      0      m(3,3) 0      0      m(3,2) 0      0      m(3,1)
          0      m(1,2) 0      m(1,1) 0      0      m(1,3) 0      0
          m(2,1) 0      0      0      m(2,2) 0      0      m(2,3) 0
          0      0      m(2,3) 0      0      m(2,2) 0      0      m(2,1)
          0      m(3,2) 0      m(3,1) 0      0      m(3,3) 0      0
          m(3,1) 0      0      0      m(3,2) 0      0      m(3,3) 0
          0      0      m(1,3) 0      0      m(1,2) 0      0      m(1,1) ];
elseif s==5
    M = [ m(1,1) 0      0      0      m(1,2)
          0      m(2,2) 0      m(2,1) 0
          0      0      m(3,3) 0      0
          0      m(1,2) 0      m(1,1) 0
          m(2,1) 0      0      0      m(2,2) ];
elseif s==4
    M = [ m(1,1) 0      0      m(1,2)
          0      m(2,2) m(2,1) 0
          0      m(1,2) m(1,1) 0
          m(2,1) 0      0      m(2,2) ];
end;
%*****
```

## Stiffness and compliance matrices

[back to index](#)

# General orthotropic stiffness matrix

$$\underline{\underline{\sigma}} = \begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{33} \\ \sigma_{12} \\ \sigma_{23} \\ \sigma_{31} \end{bmatrix} = \begin{bmatrix} A & Q & R & 0 & 0 & 0 \\ Q & B & S & 0 & 0 & 0 \\ R & S & C & 0 & 0 & 0 \\ 0 & 0 & 0 & K & 0 & 0 \\ 0 & 0 & 0 & 0 & L & 0 \\ 0 & 0 & 0 & 0 & 0 & M \end{bmatrix} \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \varepsilon_{33} \\ \gamma_{12} \\ \gamma_{23} \\ \gamma_{31} \end{bmatrix} = \underline{\underline{C}}_{\varepsilon}$$

$$\underline{\underline{\varepsilon}} = \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \varepsilon_{33} \\ \gamma_{12} \\ \gamma_{23} \\ \gamma_{31} \end{bmatrix} = \begin{bmatrix} a & q & r & 0 & 0 & 0 \\ q & b & s & 0 & 0 & 0 \\ r & s & c & 0 & 0 & 0 \\ 0 & 0 & 0 & k & 0 & 0 \\ 0 & 0 & 0 & 0 & l & 0 \\ 0 & 0 & 0 & 0 & 0 & m \end{bmatrix} \begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{33} \\ \sigma_{12} \\ \sigma_{23} \\ \sigma_{31} \end{bmatrix} = \underline{\underline{C}}^{-1} \underline{\underline{\sigma}} = \underline{\underline{S}} \underline{\underline{\sigma}}$$

## General orthotropic compliance matrix

$$\underline{\underline{C}}^{-1} = \frac{1}{\Delta_c} \begin{bmatrix} BC - S^2 & -QC + RS & QS - BR & 0 & 0 & 0 \\ -QC + RS & AC - R^2 & -AS + QR & 0 & 0 & 0 \\ QS - BR & -AS + QR & AB - Q^2 & 0 & 0 & 0 \\ 0 & 0 & 0 & \Delta_c(1/K) & 0 & 0 \\ 0 & 0 & 0 & 0 & \Delta_c(1/L) & 0 \\ 0 & 0 & 0 & 0 & 0 & \Delta_c(1/M) \end{bmatrix}$$

with  $\Delta_c = ABC - AS^2 - BR^2 - CQ^2 + 2QRS$

$$\underline{\underline{S}}^{-1} = \frac{1}{\Delta_s} \begin{bmatrix} bc - s^2 & -qc + rs & qs - br & 0 & 0 & 0 \\ -qc + rs & ac - r^2 & -as + qr & 0 & 0 & 0 \\ qs - br & -as + qr & ab - q^2 & 0 & 0 & 0 \\ 0 & 0 & 0 & \Delta_s(1/k) & 0 & 0 \\ 0 & 0 & 0 & 0 & \Delta_s(1/l) & 0 \\ 0 & 0 & 0 & 0 & 0 & \Delta_s(1/m) \end{bmatrix}$$

with  $\Delta_s = abc - as^2 - br^2 - cq^2 + 2qrs$

# Material symmetry

quadratic

$$B = A ; S = R ; M = L ;$$

transversal isotropic

$$B = A ; S = R ; M = L ; K = \frac{1}{2}(A - Q)$$

cubic

$$C = B = A ; S = R = Q ; M = L = K \neq \frac{1}{2}(A - Q)$$

isotropic

$$C = B = A ; S = R = Q ; M = L = K = \frac{1}{2}(A - Q)$$

# Orthotropic thermo-elasticity

$$\begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{33} \\ \sigma_{12} \\ \sigma_{23} \\ \sigma_{31} \end{bmatrix} = \begin{bmatrix} A & Q & R & 0 & 0 & 0 \\ Q & B & S & 0 & 0 & 0 \\ R & S & C & 0 & 0 & 0 \\ 0 & 0 & 0 & K & 0 & 0 \\ 0 & 0 & 0 & 0 & L & 0 \\ 0 & 0 & 0 & 0 & 0 & M \end{bmatrix} \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \varepsilon_{33} \\ \gamma_{12} \\ \gamma_{23} \\ \gamma_{31} \end{bmatrix} - \alpha \Delta T \begin{bmatrix} A + Q + R \\ Q + B + S \\ R + S + C \\ 0 \\ 0 \\ 0 \end{bmatrix}$$

$$\begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \varepsilon_{33} \\ \gamma_{12} \\ \gamma_{23} \\ \gamma_{31} \end{bmatrix} = \begin{bmatrix} a & q & r & 0 & 0 & 0 \\ q & b & s & 0 & 0 & 0 \\ r & s & c & 0 & 0 & 0 \\ 0 & 0 & 0 & k & 0 & 0 \\ 0 & 0 & 0 & 0 & l & 0 \\ 0 & 0 & 0 & 0 & 0 & m \end{bmatrix} \begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{33} \\ \sigma_{12} \\ \sigma_{23} \\ \sigma_{31} \end{bmatrix} + \alpha \Delta T \begin{bmatrix} 1 \\ 1 \\ 1 \\ 0 \\ 0 \\ 0 \end{bmatrix}$$

## Plane strain

$$\varepsilon_{33} = \gamma_{23} = \gamma_{31} = 0 \quad \rightarrow \quad \sigma_{33} = R\varepsilon_{11} + S\varepsilon_{22}$$

$$\underline{\underline{\sigma}} = \begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{12} \end{bmatrix} = \begin{bmatrix} A & Q & 0 \\ Q & B & 0 \\ 0 & 0 & K \end{bmatrix} \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \gamma_{12} \end{bmatrix} = \begin{bmatrix} A_{\varepsilon} & Q_{\varepsilon} & 0 \\ Q_{\varepsilon} & B_{\varepsilon} & 0 \\ 0 & 0 & K \end{bmatrix} \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \gamma_{12} \end{bmatrix} = \underline{\underline{C}}_{\varepsilon} \underline{\underline{\varepsilon}}$$

$$\begin{aligned} \underline{\underline{\varepsilon}} = \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \gamma_{12} \end{bmatrix} &= \frac{1}{AB - Q^2} \begin{bmatrix} B & -Q & 0 \\ -Q & A & 0 \\ 0 & 0 & \frac{AB - Q^2}{K} \end{bmatrix} \begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{12} \end{bmatrix} \\ &= \begin{bmatrix} a_{\varepsilon} & q_{\varepsilon} & 0 \\ q_{\varepsilon} & b_{\varepsilon} & 0 \\ 0 & 0 & k \end{bmatrix} \begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{12} \end{bmatrix} = \underline{\underline{S}}_{\varepsilon} \underline{\underline{\sigma}} \end{aligned}$$

# Plane strain

$$\varepsilon_{33} = 0 = r\sigma_{11} + s\sigma_{22} + c\sigma_{33} \quad \rightarrow \quad \sigma_{33} = -\frac{r}{c}\sigma_{11} - \frac{s}{c}\sigma_{22}$$

$$\underline{\underline{\varepsilon}} = \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \gamma_{12} \end{bmatrix} = \begin{bmatrix} a & q & 0 \\ q & b & 0 \\ 0 & 0 & k \end{bmatrix} \begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{12} \end{bmatrix} - \begin{bmatrix} r \\ s \\ 0 \end{bmatrix} \begin{bmatrix} \frac{r}{c} & \frac{s}{c} & 0 \end{bmatrix} \begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{12} \end{bmatrix}$$

$$= \frac{1}{c} \begin{bmatrix} ac - r^2 & qc - rs & 0 \\ qc - sr & bc - s^2 & 0 \\ 0 & 0 & kc \end{bmatrix} \begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{12} \end{bmatrix} = \begin{bmatrix} a_{\varepsilon} & q_{\varepsilon} & 0 \\ q_{\varepsilon} & b_{\varepsilon} & 0 \\ 0 & 0 & k \end{bmatrix} \begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{12} \end{bmatrix} = \underline{\underline{S}}_{\varepsilon} \underline{\underline{\sigma}}$$

$$\underline{\underline{\sigma}} = \begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{12} \end{bmatrix} = \begin{bmatrix} a_{\varepsilon} & q_{\varepsilon} & 0 \\ q_{\varepsilon} & b_{\varepsilon} & 0 \\ 0 & 0 & k \end{bmatrix}^{-1} \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \gamma_{12} \end{bmatrix} = \frac{1}{\Delta_s} \begin{bmatrix} bc - s^2 & -qc + rs & 0 \\ -qc + rs & ac - r^2 & 0 \\ 0 & 0 & \frac{\Delta_s}{k} \end{bmatrix} \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \varepsilon_{12} \end{bmatrix}$$

$$\text{with } \Delta_s = abc - as^2 - br^2 - cq^2 + 2qrs$$

$$= \begin{bmatrix} A_{\varepsilon} & Q_{\varepsilon} & 0 \\ Q_{\varepsilon} & B_{\varepsilon} & 0 \\ 0 & 0 & K \end{bmatrix} \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \varepsilon_{12} \end{bmatrix} = \underline{\underline{C}}_{\varepsilon} \underline{\underline{\sigma}}$$

$$\sigma_{33} = -\frac{1}{\Delta_s} [(br - qs)\varepsilon_{11} + (as - qr)\varepsilon_{22}]$$



# Plane stress

$$\sigma_{33} = \sigma_{23} = \sigma_{31} = 0 \quad \rightarrow \quad \varepsilon_{33} = r\sigma_{11} + s\sigma_{22}$$

$$\underline{\underline{\varepsilon}} = \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \gamma_{12} \end{bmatrix} = \begin{bmatrix} a & q & 0 \\ q & b & 0 \\ 0 & 0 & k \end{bmatrix} \begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{12} \end{bmatrix} = \begin{bmatrix} a_{\sigma} & q_{\sigma} & 0 \\ q_{\sigma} & b_{\sigma} & 0 \\ 0 & 0 & k \end{bmatrix} \begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{12} \end{bmatrix} = \underline{\underline{S}}_{\sigma} \underline{\underline{\sigma}}$$

$$\underline{\underline{\sigma}} = \begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{12} \end{bmatrix} = \frac{1}{ab - q^2} \begin{bmatrix} b & -q & 0 \\ -q & a & 0 \\ 0 & 0 & \frac{ab - q^2}{k} \end{bmatrix} \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \gamma_{12} \end{bmatrix}$$

$$= \begin{bmatrix} A_{\sigma} & Q_{\sigma} & 0 \\ Q_{\sigma} & B_{\sigma} & 0 \\ 0 & 0 & K \end{bmatrix} \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \gamma_{12} \end{bmatrix} = \underline{\underline{C}}_{\sigma} \underline{\underline{\varepsilon}}$$

$$\varepsilon_{33} = \frac{1}{ab - q^2} [(br - qs)\varepsilon_{11} + (as - qr)\varepsilon_{22}]$$

# Plane stress

$$\sigma_{33} = 0 = R\varepsilon_{11} + S\varepsilon_{22} + C\varepsilon_{33} \rightarrow \varepsilon_{33} = -\frac{R}{C}\varepsilon_{11} - \frac{S}{C}\varepsilon_{22}$$

$$\underline{\sigma} = \begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{12} \end{bmatrix} = \begin{bmatrix} A & Q & 0 \\ Q & B & 0 \\ 0 & 0 & K \end{bmatrix} \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \gamma_{12} \end{bmatrix} - \begin{bmatrix} R \\ S \\ 0 \end{bmatrix} \begin{bmatrix} \frac{R}{C} & \frac{S}{C} & 0 \end{bmatrix} \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \gamma_{12} \end{bmatrix}$$

$$= \frac{1}{C} \begin{bmatrix} AC - R^2 & QC - RS & 0 \\ QC - SR & BC - S^2 & 0 \\ 0 & 0 & KC \end{bmatrix} \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \gamma_{12} \end{bmatrix}$$

$$= \begin{bmatrix} A_{\sigma} & Q_{\sigma} & 0 \\ Q_{\sigma} & B_{\sigma} & 0 \\ 0 & 0 & K \end{bmatrix} \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \gamma_{12} \end{bmatrix} = \underline{\underline{C}}_{\sigma} \underline{\varepsilon}$$

$$\underline{\varepsilon} = \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \gamma_{12} \end{bmatrix} = \begin{bmatrix} A_{\sigma} & Q_{\sigma} & 0 \\ Q_{\sigma} & B_{\sigma} & 0 \\ 0 & 0 & K \end{bmatrix}^{-1} \begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{12} \end{bmatrix} = \begin{bmatrix} a_{\sigma} & q_{\sigma} & 0 \\ q_{\sigma} & b_{\sigma} & 0 \\ 0 & 0 & k \end{bmatrix} \begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{12} \end{bmatrix} = \underline{\underline{S}}_{\sigma} \underline{\sigma}$$

# General planar material laws

$$\underline{\underline{C}}_p = \begin{bmatrix} A_p & Q_p & 0 \\ Q_p & B_p & 0 \\ 0 & 0 & K \end{bmatrix} - \alpha \Delta T \begin{bmatrix} \Theta_{p1} \\ \Theta_{p2} \\ 0 \end{bmatrix}$$

$$\underline{\underline{S}}_p = \begin{bmatrix} a_p & q_p & 0 \\ q_p & b_p & 0 \\ 0 & 0 & k \end{bmatrix} + \sigma \Delta T \begin{bmatrix} \theta_{p1} \\ \theta_{p2} \\ 0 \end{bmatrix}$$

plane strain :  $(\ )_p = (\ )_\varepsilon$

plane stress :  $(\ )_p = (\ )_\sigma$

# Linear elastic orthotropic, 3D

$$\underline{\underline{S}} = \begin{bmatrix} E_1^{-1} & -\nu_{21}E_2^{-1} & -\nu_{31}E_3^{-1} & 0 & 0 & 0 \\ -\nu_{12}E_1^{-1} & E_2^{-1} & -\nu_{32}E_3^{-1} & 0 & 0 & 0 \\ -\nu_{13}E_1^{-1} & -\nu_{23}E_2^{-1} & E_3^{-1} & 0 & 0 & 0 \\ 0 & 0 & 0 & G_{12}^{-1} & 0 & 0 \\ 0 & 0 & 0 & 0 & G_{23}^{-1} & 0 \\ 0 & 0 & 0 & 0 & 0 & G_{31}^{-1} \end{bmatrix}$$

$$\text{with} \quad \frac{\nu_{12}}{E_1} = \frac{\nu_{21}}{E_2} \quad ; \quad \frac{\nu_{23}}{E_2} = \frac{\nu_{32}}{E_3} \quad ; \quad \frac{\nu_{31}}{E_3} = \frac{\nu_{13}}{E_1}$$

$$\underline{\underline{C}} = \frac{1}{\Delta_s} \begin{bmatrix} \frac{1-\nu_{32}\nu_{23}}{E_2E_3} & \frac{\nu_{31}\nu_{23}+\nu_{21}}{E_2E_3} & \frac{\nu_{21}\nu_{32}+\nu_{31}}{E_2E_3} & 0 & 0 & 0 \\ \frac{\nu_{13}\nu_{32}+\nu_{12}}{E_1E_3} & \frac{1-\nu_{31}\nu_{13}}{E_1E_3} & \frac{\nu_{12}\nu_{31}+\nu_{32}}{E_1E_3} & 0 & 0 & 0 \\ \frac{\nu_{12}\nu_{23}+\nu_{13}}{E_1E_2} & \frac{\nu_{21}\nu_{13}+\nu_{23}}{E_1E_2} & \frac{1-\nu_{12}\nu_{21}}{E_1E_2} & 0 & 0 & 0 \\ 0 & 0 & 0 & \Delta_s G_{12} & 0 & 0 \\ 0 & 0 & 0 & 0 & \Delta_s G_{23} & 0 \\ 0 & 0 & 0 & 0 & 0 & \Delta_s G_{31} \end{bmatrix}$$

$$\text{with} \quad \Delta_s = \frac{1 - \nu_{12}\nu_{21} - \nu_{23}\nu_{32} - \nu_{31}\nu_{13} - \nu_{12}\nu_{23}\nu_{31} - \nu_{21}\nu_{32}\nu_{13}}{E_1E_2E_3}$$

# Voigt notation

$$\underline{\sigma}^T = [\sigma_{11} \ \sigma_{22} \ \sigma_{33} \ \sigma_{12} \ \sigma_{23} \ \sigma_{31}] = [\sigma_1 \ \sigma_2 \ \sigma_3 \ \sigma_6 \ \sigma_4 \ \sigma_5]$$

$$\underline{\varepsilon}^T = [\varepsilon_{11} \ \varepsilon_{22} \ \varepsilon_{33} \ \gamma_{12} \ \gamma_{23} \ \gamma_{31}] = [\varepsilon_1 \ \varepsilon_2 \ \varepsilon_3 \ \varepsilon_6 \ \varepsilon_4 \ \varepsilon_5]$$

$$\begin{bmatrix} \varepsilon_1 \\ \varepsilon_2 \\ \varepsilon_3 \\ \varepsilon_4 \\ \varepsilon_5 \\ \varepsilon_6 \end{bmatrix} = \begin{bmatrix} S_{11} & S_{12} & S_{13} & 0 & 0 & 0 \\ S_{21} & S_{22} & S_{23} & 0 & 0 & 0 \\ S_{31} & S_{32} & S_{33} & 0 & 0 & 0 \\ 0 & 0 & 0 & S_{44} & 0 & 0 \\ 0 & 0 & 0 & 0 & S_{55} & 0 \\ 0 & 0 & 0 & 0 & 0 & S_{66} \end{bmatrix} \begin{bmatrix} \sigma_1 \\ \sigma_2 \\ \sigma_3 \\ \sigma_4 \\ \sigma_5 \\ \sigma_6 \end{bmatrix}$$

## Linear elastic orthotropic, plane strain

$$\sigma_{33} = \nu_{13} \frac{E_3}{E_1} \sigma_{11} + \nu_{23} \frac{E_3}{E_2} \sigma_{22}$$

$$\underline{\underline{S}}_{\varepsilon} = \begin{bmatrix} \frac{1-\nu_{31}\nu_{13}}{E_1} & -\frac{\nu_{31}\nu_{23}+\nu_{21}}{E_2} & 0 \\ -\frac{\nu_{13}\nu_{32}+\nu_{12}}{E_1} & \frac{1-\nu_{32}\nu_{23}}{E_2} & 0 \\ 0 & 0 & \frac{1}{G_{12}} \end{bmatrix}$$

$$\underline{\underline{C}}_{\varepsilon} = \underline{\underline{S}}_{\varepsilon}^{-1} = \frac{1}{\Delta_s} \begin{bmatrix} \frac{1-\nu_{32}\nu_{23}}{E_2 E_3} & \frac{\nu_{31}\nu_{23}+\nu_{21}}{E_2 E_3} & 0 \\ \frac{\nu_{13}\nu_{32}+\nu_{12}}{E_1 E_3} & \frac{1-\nu_{31}\nu_{13}}{E_1 E_3} & 0 \\ 0 & 0 & \Delta_s G_{12} \end{bmatrix}$$

$$\text{with } \Delta_s = \frac{1 - \nu_{12}\nu_{21} - \nu_{23}\nu_{32} - \nu_{31}\nu_{13} - \nu_{12}\nu_{23}\nu_{31} - \nu_{21}\nu_{32}\nu_{13}}{E_1 E_2 E_3}$$

$$\sigma_{33} = \frac{1}{\Delta_s} \left\{ \frac{\nu_{12}\nu_{32} + \nu_{13}}{E_1 E_2} \varepsilon_{11} + \frac{\nu_{21}\nu_{13} + \nu_{23}}{E_1 E_2} \varepsilon_{22} \right\}$$

## Linear elastic orthotropic, plane stress

$$\varepsilon_{33} = -\nu_{13}E_1^{-1}\sigma_{11} - \nu_{23}E_2^{-1}\sigma_{22}$$

$$\underline{\underline{S}}_{\sigma} = \begin{bmatrix} E_1^{-1} & -\nu_{21}E_2^{-1} & 0 \\ -\nu_{12}E_1^{-1} & E_2^{-1} & 0 \\ 0 & 0 & G_{12}^{-1} \end{bmatrix}$$

$$\underline{\underline{C}}_{\sigma} = \underline{\underline{S}}_{\sigma}^{-1} = \frac{1}{1 - \nu_{21}\nu_{12}} \begin{bmatrix} E_1 & \nu_{21}E_1 & 0 \\ \nu_{12}E_2 & E_2 & 0 \\ 0 & 0 & (1 - \nu_{21}\nu_{12})G_{12} \end{bmatrix}$$

$$\varepsilon_{33} = -\frac{1}{1 - \nu_{12}\nu_{21}} \{(\nu_{12}\nu_{23} + \nu_{13})\varepsilon_{11} + (\nu_{21}\nu_{13} + \nu_{23})\varepsilon_{22}\}$$

## Linear elastic transversal isotropic, 3D

$$\underline{\underline{S}} = \begin{bmatrix} E_p^{-1} & -\nu_p E_p^{-1} & -\nu_{3p} E_3^{-1} & 0 & 0 & 0 \\ -\nu_p E_p^{-1} & E_p^{-1} & -\nu_{3p} E_3^{-1} & 0 & 0 & 0 \\ -\nu_{p3} E_p^{-1} & -\nu_{p3} E_p^{-1} & E_3^{-1} & 0 & 0 & 0 \\ 0 & 0 & 0 & G_p^{-1} & 0 & 0 \\ 0 & 0 & 0 & 0 & G_{p3}^{-1} & 0 \\ 0 & 0 & 0 & 0 & 0 & G_{3p}^{-1} \end{bmatrix}$$

with  $\frac{\nu_{p3}}{E_p} = \frac{\nu_{3p}}{E_3}$

$$\underline{\underline{C}} = \underline{\underline{S}}^{-1} = \frac{1}{\Delta_s} \begin{bmatrix} \frac{1-\nu_{3p}\nu_{p3}}{E_p E_3} & \frac{\nu_{3p}\nu_{p3}+\nu_p}{E_p E_3} & \frac{\nu_p\nu_{3p}+\nu_{3p}}{E_p E_3} & 0 & 0 & 0 \\ \frac{\nu_{p3}\nu_{3p}+\nu_p}{E_p E_3} & \frac{1-\nu_{3p}\nu_{p3}}{E_p E_3} & \frac{\nu_p\nu_{3p}+\nu_{3p}}{E_p E_3} & 0 & 0 & 0 \\ \frac{\nu_p\nu_{p3}+\nu_{p3}}{E_p E_p} & \frac{\nu_p\nu_{p3}+\nu_{p3}}{E_p E_p} & \frac{1-\nu_p\nu_p}{E_p E_p} & 0 & 0 & 0 \\ 0 & 0 & 0 & \Delta_s G_p & 0 & 0 \\ 0 & 0 & 0 & 0 & \Delta_s G_{p3} & 0 \\ 0 & 0 & 0 & 0 & 0 & \Delta_s G_{3p} \end{bmatrix}$$

with  $\Delta_s = \frac{1 - \nu_p\nu_p - \nu_{p3}\nu_{3p} - \nu_{3p}\nu_{p3} - \nu_p\nu_{p3}\nu_{3p} - \nu_p\nu_{3p}\nu_{p3}}{E_p E_p E_3}$



## Linear elastic transversal isotropic, plane strain

$$\sigma_{33} = \frac{E_3 \nu_{p3}}{E_p} (\sigma_{11} + \sigma_{22}) = \nu_{3p} (\sigma_{11} + \sigma_{22})$$

$$\underline{\underline{S}}_{\varepsilon} = \begin{bmatrix} \frac{1 - \nu_{3p} \nu_{p3}}{E_p} & -\frac{\nu_{3p} \nu_{p3} + \nu_p}{E_p} & 0 \\ -\frac{\nu_{p3} \nu_{3p} + \nu_p}{E_p} & \frac{1 - \nu_{3p} \nu_{p3}}{E_p} & 0 \\ 0 & 0 & \frac{1}{G_p} \end{bmatrix}$$

$$\underline{\underline{C}}_{\varepsilon} = \underline{\underline{S}}_{\varepsilon}^{-1} = \frac{1}{\Delta_s} \begin{bmatrix} \frac{1 - \nu_{3p} \nu_{p3}}{E_p E_3} & \frac{\nu_{3p} \nu_{p3} + \nu_p}{E_p E_3} & 0 \\ \frac{\nu_{p3} \nu_{3p} + \nu_p}{E_p E_3} & \frac{1 - \nu_{3p} \nu_{p3}}{E_p E_3} & 0 \\ 0 & 0 & \Delta_s G_p \end{bmatrix}$$

$$\text{with } \Delta_s = \frac{1 - \nu_p \nu_p - \nu_{p3} \nu_{3p} - \nu_{3p} \nu_{p3} - \nu_p \nu_{p3} \nu_{3p} - \nu_p \nu_{3p} \nu_{p3}}{E_p E_p E_3}$$

$$\sigma_{33} = \frac{1}{\Delta_s} \frac{\nu_{p3} (\nu_p + 1)}{E_p^2} (\varepsilon_{11} + \varepsilon_{22})$$

## Linear elastic transversal isotropic, plane stress

$$\varepsilon_{33} = -\frac{\nu_{p3}}{E_p}(\sigma_{11} + \sigma_{22})$$

$$\underline{\underline{S}}_{\sigma} = \begin{bmatrix} E_p^{-1} & -\nu_p E_p^{-1} & 0 \\ -\nu_p E_p^{-1} & E_p^{-1} & 0 \\ 0 & 0 & G_p^{-1} \end{bmatrix}$$

$$\underline{\underline{C}}_{\sigma} = \underline{\underline{S}}_{\sigma}^{-1} = \frac{1}{1 - \nu_p \nu_p} \begin{bmatrix} E_p & \nu_p E_p & 0 \\ \nu_p E_p & E_p & 0 \\ 0 & 0 & (1 - \nu_p \nu_p) G_p \end{bmatrix}$$

$$\varepsilon_{33} = -\frac{\nu_{p3}}{1 - \nu_p}(\varepsilon_{11} + \varepsilon_{22})$$

## Linear elastic isotropic, 3D

$$\underline{\underline{S}} = \frac{1}{E} \begin{bmatrix} 1 & -\nu & -\nu & 0 & 0 & 0 \\ -\nu & 1 & -\nu & 0 & 0 & 0 \\ -\nu & -\nu & 1 & 0 & 0 & 0 \\ 0 & 0 & 0 & 2(1+\nu) & 0 & 0 \\ 0 & 0 & 0 & 0 & 2(1+\nu) & 0 \\ 0 & 0 & 0 & 0 & 0 & 2(1+\nu) \end{bmatrix}$$

$$\underline{\underline{C}} = \underline{\underline{S}}^{-1} = \frac{E}{(1+\nu)(1-2\nu)} \begin{bmatrix} 1-\nu & \nu & \nu & 0 & 0 & 0 \\ \nu & 1-\nu & \nu & 0 & 0 & 0 \\ \nu & \nu & 1-\nu & 0 & 0 & 0 \\ 0 & 0 & 0 & \frac{1}{2}(1-2\nu) & 0 & 0 \\ 0 & 0 & 0 & 0 & \frac{1}{2}(1-2\nu) & 0 \\ 0 & 0 & 0 & 0 & 0 & \frac{1}{2}(1-2\nu) \end{bmatrix}$$

## Linear elastic isotropic, plane strain

$$\sigma_{33} = \nu(\sigma_{11} + \sigma_{22})$$

$$\underline{\underline{S}}_{\varepsilon} = \frac{1+\nu}{E} \begin{bmatrix} 1-\nu & -\nu & 0 \\ -\nu & 1-\nu & 0 \\ 0 & 0 & 2 \end{bmatrix}$$

$$\underline{\underline{C}}_{\varepsilon} = \underline{\underline{S}}_{\varepsilon}^{-1} = \frac{E}{(1+\nu)(1-2\nu)} \begin{bmatrix} 1-\nu & \nu & 0 \\ \nu & 1-\nu & 0 \\ 0 & 0 & \frac{1}{2}(1-2\nu) \end{bmatrix}$$

$$\sigma_{33} = \frac{E}{(1+\nu)(1-2\nu)} \nu(\varepsilon_{11} + \varepsilon_{22})$$

## Linear elastic isotropic, plane stress

$$\varepsilon_{33} = -\frac{\nu}{E} (\sigma_{11} + \sigma_{22})$$

$$\underline{\underline{S}}_{\sigma} = \frac{1}{E} \begin{bmatrix} 1 & -\nu & 0 \\ -\nu & 1 & 0 \\ 0 & 0 & 2(1+\nu) \end{bmatrix}$$

$$\underline{\underline{C}}_{\sigma} = \underline{\underline{S}}_{\sigma}^{-1} = \frac{E}{1-\nu^2} \begin{bmatrix} 1 & \nu & 0 \\ \nu & 1 & 0 \\ 0 & 0 & \frac{1}{2}(1-\nu) \end{bmatrix}$$

$$\varepsilon_{33} = -\frac{\nu}{1-\nu} (\varepsilon_{11} + \varepsilon_{22})$$

## WR for axi-symmetric deformation

[back to index](#)

## Weighted residual formulation for axi-symmetric deformation

$$\sigma_{rr,r} + \frac{1}{r}(\sigma_{rr} - \sigma_{tt}) + q_r = 0 \quad \forall \quad r \quad \leftrightarrow$$

$$\int_V w \left\{ \sigma_{rr,r} + \frac{1}{r}(\sigma_{rr} - \sigma_{tt}) + q_r \right\} dV = 0 \quad \forall \quad w(r)$$

$$2\pi t \int_{R_i}^{R_o} w \left\{ \sigma_{rr,r} + \frac{1}{r}(\sigma_{rr} - \sigma_{tt}) + q_r \right\} r dr = 0 \quad \forall \quad w(r)$$

## Weak formulation

$$w\sigma_{rr,r}r = w\frac{d\sigma_{rr}}{dr}r = \frac{d}{dr}(w\sigma_{rr}r) - \frac{dw}{dr}\sigma_{rr}r - w\sigma_{rr} \quad \rightarrow$$
$$\int_{R_i}^{R_o} (w_{,r}\sigma_{rr}r + w\sigma_{tt}) dr = \int_{R_i}^{R_o} wq_r r dr + [w\sigma_{rr}t]_{R_i}^{R_o} = f_e$$



## Linear elastic deformation

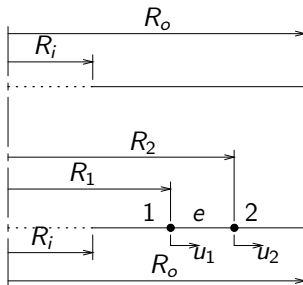
$$\left. \begin{aligned} \sigma_{rr} &= A_p \varepsilon_{rr} + Q_p \varepsilon_{tt} = A_p u_{r,r} + Q_p \frac{u_r}{r} \\ \sigma_{tt} &= Q_p \varepsilon_{rr} + B_p \varepsilon_{tt} = Q_p u_{r,r} + B_p \frac{u_r}{r} \end{aligned} \right\} \rightarrow$$
$$\int_{R_i}^{R_o} \left\{ w_{,r} \left( A_p u_{r,r} + Q_p \frac{u_r}{r} \right) r + w \left( Q_p u_{r,r} + B_p \frac{u_r}{r} \right) \right\} dr = f_e$$

## FEM for axi-symmetric deformation

[back to index](#)

# Finite element method for an axi-symmetric ring

# Discretisation



$$\sum_{e=1}^{ne} \int_{R_1}^{R_2} \left[ A_p w_{,r} u_{r,r} + Q_p w_{,r} u_r + Q_p w u_{r,r} + B_p w \frac{1}{r} u_r \right] dr = \sum_{e=1}^{ne} f_e^e$$

# Interpolation

$$u_r = \psi_1 u_1 + \psi_2 u_2$$

$$\text{Galerkin} \quad \rightarrow \quad w = \psi_1 w_1 + \psi_2 w_2$$

# Substitution

$$\begin{aligned}
 & \begin{bmatrix} w_1 & w_2 \end{bmatrix} \int_{R_1}^{R_2} \left\{ A_p \begin{bmatrix} \psi_{1,r} \\ \psi_{2,r} \end{bmatrix} \begin{bmatrix} \psi_{1,r} & \psi_{2,r} \end{bmatrix} r + Q_p \begin{bmatrix} \psi_{1,r} \\ \psi_{2,r} \end{bmatrix} \begin{bmatrix} \psi_1 & \psi_2 \end{bmatrix} + \right. \\
 & \quad \left. Q_p \begin{bmatrix} \psi_1 \\ \psi_2 \end{bmatrix} \begin{bmatrix} \psi_{1,r} & \psi_{2,r} \end{bmatrix} + B_p \begin{bmatrix} \psi_1 \\ \psi_2 \end{bmatrix} \begin{bmatrix} \psi_1 & \psi_2 \end{bmatrix} \frac{1}{r} \right\} \begin{bmatrix} u_{r1} \\ u_{r2} \end{bmatrix} = \begin{bmatrix} w_1 & w_2 \end{bmatrix} \underline{f}_e^e \\
 & \begin{bmatrix} w_1 & w_2 \end{bmatrix} \left\{ A_p \begin{bmatrix} \psi_{1,r}\psi_{1,r} & \psi_{1,r}\psi_{2,r} \\ \psi_{2,r}\psi_{1,r} & \psi_{2,r}\psi_{2,r} \end{bmatrix} r + Q_p \begin{bmatrix} \psi_{1,r}\psi_1 & \psi_{1,r}\psi_2 \\ \psi_{2,r}\psi_1 & \psi_{2,r}\psi_2 \end{bmatrix} + \right. \\
 & \quad \left. Q_p \begin{bmatrix} \psi_1\psi_{1,r} & \psi_1\psi_{2,r} \\ \psi_2\psi_{1,r} & \psi_2\psi_{2,r} \end{bmatrix} + B_p \begin{bmatrix} \psi_1\psi_1 & \psi_1\psi_2 \\ \psi_2\psi_1 & \psi_2\psi_2 \end{bmatrix} \frac{1}{r} \right\} dr \begin{bmatrix} u_{r1} \\ u_{r2} \end{bmatrix} = \begin{bmatrix} w_1 & w_2 \end{bmatrix} \underline{f}_e^e \\
 & \underline{w}^{eT} \underline{K}^e \underline{u}^e = \underline{w}^{eT} \underline{f}_e^e
 \end{aligned}$$

# Integration

$$K_{11}^e = \int_{R_i}^{R_o} \left[ A_p \psi_{1,r} \psi_{1,r} r + Q_p \psi_{1,r} \psi_1 + Q_p \psi_1 \psi_{1,r} + B_p \psi_1 \psi_1 \frac{1}{r} \right] dr$$

$$K_{12}^e = \int_{R_1}^{R_2} \left[ A_p \psi_{1,r} \psi_{2,r} r + Q_p \psi_{1,r} \psi_2 + Q_p \psi_1 \psi_{2,r} + B_p \psi_1 \psi_2 \frac{1}{r} \right] dr$$

$$K_{21}^e = \int_{R_1}^{R_2} \left[ A_p \psi_{2,r} \psi_{1,r} r + Q_p \psi_{2,r} \psi_1 + Q_p \psi_2 \psi_{1,r} + B_p \psi_2 \psi_1 \frac{1}{r} \right] dr$$

$$K_{22}^e = \int_{R_1}^{R_2} \left[ A_p \psi_{2,r} \psi_{2,r} r + Q_p \psi_{2,r} \psi_2 + Q_p \psi_2 \psi_{2,r} + B_p \psi_2 \psi_2 \frac{1}{r} \right] dr$$

## External load

$$f_e = \int_{R_i}^{R_o} w q_r r \, dr + [w \sigma_{rr} r]_{R_i}^{R_o} = \sum_{e=1}^{ne} \int_{R_1}^{R_2} w q_r r \, dr + [w \sigma_{rr} r]_{R_i}^{R_o} = \sum_{e=1}^{ne} q_e^e + [w \sigma_{rr} r]_{R_i}^{R_o}$$



## Volume load = centrifugal load

$$q_e^e = \rho \omega^2 \int_{R_1}^{R_2} w r^2 dr = \begin{bmatrix} w_1 & w_2 \end{bmatrix} \rho \omega^2 \int_{R_1}^{R_2} \begin{bmatrix} \psi_1 \\ \psi_2 \end{bmatrix} r^2 dr = \tilde{w}^{eT} \tilde{q}^e$$
$$f_e^e = \tilde{w}^{eT} \tilde{q}^e + w_o \sigma_{rr}(r = R_o) R_o - w_i \sigma_{rr}(r = R_i) R_i$$

# Assembling

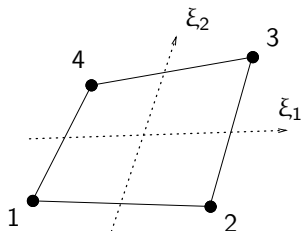
$$\tilde{w}^T \underline{K} \tilde{u} = \tilde{w}^T \tilde{f}_e \quad \forall \quad \tilde{w} \quad \Rightarrow \quad \underline{K} \tilde{u} = \tilde{f}_e$$

# Boundary conditions

## FEM for planar deformation

[back to index](#)

# Four-node quadrilateral element

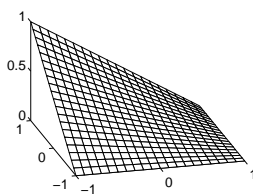
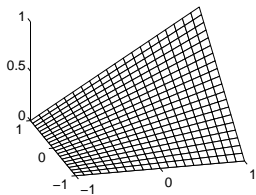
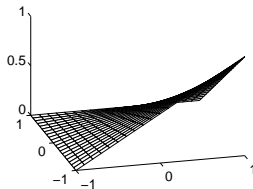
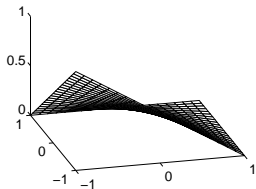


interpolation (shape) functions bi-linear in  $(\xi_1, \xi_2)$

$$N^1 = \frac{1}{4}(1 - \xi_1)(1 - \xi_2) \quad ; \quad N^2 = \frac{1}{4}(1 + \xi_1)(1 - \xi_2)$$

$$N^3 = \frac{1}{4}(1 + \xi_1)(1 + \xi_2) \quad ; \quad N^4 = \frac{1}{4}(1 - \xi_1)(1 + \xi_2)$$

# Shape functions



# Cartesian coordinate system

displacement

$$\begin{bmatrix} u_x \\ u_y \end{bmatrix} = \begin{bmatrix} N^1 & 0 & N^2 & 0 & N^3 & 0 & N^4 & 0 \\ 0 & N^1 & 0 & N^2 & 0 & N^3 & 0 & N^4 \end{bmatrix} \begin{bmatrix} u_x^1 \\ u_y^1 \\ u_x^2 \\ u_y^2 \\ u_x^3 \\ u_y^3 \\ u_x^4 \\ u_y^4 \end{bmatrix} \rightarrow \underline{u} = \underline{N} \underline{u}^e$$

element shape

$$\underline{x} = \underline{N} \underline{x}^e \quad ; \quad \underline{x}_0 = \underline{N} \underline{x}_0^e$$

weighting function

$$\underline{w} = \underline{N} \underline{w}^e$$

# Derivatives

$$\begin{bmatrix} u_{x,x} \\ u_{y,y} \\ u_{y,x} \\ u_{x,y} \end{bmatrix} = \begin{bmatrix} N_{,x}^1 & 0 & N_{,x}^2 & 0 & N_{,x}^3 & 0 & N_{,x}^4 & 0 \\ 0 & N_{,y}^1 & 0 & N_{,y}^2 & 0 & N_{,y}^3 & 0 & N_{,y}^4 \\ 0 & N_{,x}^1 & 0 & N_{,x}^2 & 0 & N_{,x}^3 & 0 & N_{,x}^4 \\ N_{,y}^1 & 0 & N_{,y}^2 & 0 & N_{,y}^3 & 0 & N_{,y}^4 & 0 \end{bmatrix} \begin{bmatrix} u_x^1 \\ u_y^1 \\ u_x^2 \\ u_y^2 \\ u_x^3 \\ u_y^3 \\ u_x^4 \\ u_y^4 \end{bmatrix} \rightarrow \left( \underline{\underline{L}}_u \right)_t = \underline{\underline{B}} \underline{u}^e$$

$$\begin{bmatrix} N_{,x}^1 & N_{,y}^1 \\ N_{,x}^2 & N_{,y}^2 \\ N_{,x}^3 & N_{,y}^3 \\ N_{,x}^4 & N_{,y}^4 \end{bmatrix} = \begin{bmatrix} N_{,1}^1 & N_{,2}^1 \\ N_{,1}^2 & N_{,2}^2 \\ N_{,1}^3 & N_{,2}^3 \\ N_{,1}^4 & N_{,2}^4 \end{bmatrix} \begin{bmatrix} \xi_{1,x} & \xi_{1,y} \\ \xi_{2,x} & \xi_{2,y} \end{bmatrix} = \begin{bmatrix} N_{,1}^1 & N_{,2}^1 \\ N_{,1}^2 & N_{,2}^2 \\ N_{,1}^3 & N_{,2}^3 \\ N_{,1}^4 & N_{,2}^4 \end{bmatrix} \underline{\underline{J}}^{-T}$$

$$\underline{\underline{J}} = \begin{bmatrix} x_{,1} & y_{,1} \\ x_{,2} & y_{,2} \end{bmatrix} = \begin{bmatrix} N_{,1}^1 & N_{,1}^2 & N_{,1}^3 & N_{,1}^4 \\ N_{,2}^1 & N_{,2}^2 & N_{,2}^3 & N_{,2}^4 \end{bmatrix} \begin{bmatrix} x_e^1 & y_e^1 \\ x_e^2 & y_e^2 \\ x_e^3 & y_e^3 \\ x_e^4 & y_e^4 \end{bmatrix}$$



# Deformation matrix

$$\underline{F} = \begin{bmatrix} \frac{\partial x}{\partial x_0} & \frac{\partial x}{\partial y_0} & 0 \\ \frac{\partial y}{\partial x_0} & \frac{\partial y}{\partial y_0} & 0 \\ 0 & 0 & F_{zz} \end{bmatrix}$$

$$\begin{aligned} \begin{bmatrix} \frac{\partial x}{\partial x_0} & \frac{\partial y}{\partial x_0} \\ \frac{\partial x}{\partial y_0} & \frac{\partial y}{\partial y_0} \end{bmatrix} &= \begin{bmatrix} N_{,x0}^1 & N_{,x0}^2 & N_{,x0}^3 & N_{,x0}^4 \\ N_{,y0}^1 & N_{,y0}^2 & N_{,y0}^3 & N_{,y0}^4 \end{bmatrix} \begin{bmatrix} x_e^1 & y_e^1 \\ x_e^2 & y_e^2 \\ x_e^3 & y_e^3 \\ x_e^4 & y_e^4 \end{bmatrix} \\ &= \begin{bmatrix} \xi_{1,x0} & \xi_{2,x0} \\ \xi_{1,y0} & \xi_{2,y0} \end{bmatrix} \begin{bmatrix} N_{,2}^1 & N_{,2}^2 & N_{,2}^3 & N_{,2}^4 \\ N_{,2}^1 & N_{,2}^2 & N_{,2}^3 & N_{,2}^4 \end{bmatrix} \begin{bmatrix} x_e^1 & y_e^1 \\ x_e^2 & y_e^2 \\ x_e^3 & y_e^3 \\ x_e^4 & y_e^4 \end{bmatrix} \\ &= \underline{J}_0^{-1} \underline{J} \end{aligned}$$

# Cylindrical coordinate system

displacement

$$\begin{bmatrix} u_r \\ u_z \end{bmatrix} = \begin{bmatrix} N^1 & 0 & N^2 & 0 & N^3 & 0 & N^4 & 0 \\ 0 & N^1 & 0 & N^2 & 0 & N^3 & 0 & N^4 \end{bmatrix} \begin{bmatrix} u_r^1 \\ u_z^1 \\ u_r^2 \\ u_z^2 \\ u_r^3 \\ u_z^3 \\ u_r^4 \\ u_z^4 \end{bmatrix} \rightarrow \underline{u} = \underline{N} \underline{u}_e$$

element shape

$$r = \underline{\tilde{N}}^T \underline{\tilde{r}} \quad ; \quad z = \underline{\tilde{N}}^T \underline{\tilde{z}}$$

$$r_0 = \underline{\tilde{N}}^T \underline{\tilde{r}}_0 \quad ; \quad z_0 = \underline{\tilde{N}}^T \underline{\tilde{z}}_0$$

weighting function

$$\underline{\tilde{w}} = \underline{N} \underline{\tilde{w}}^e$$

# Derivatives

$$\begin{bmatrix} u_{r,r} \\ u_{z,z} \\ \frac{1}{r}u_r \\ u_{z,r} \\ u_{r,z} \end{bmatrix} = \begin{bmatrix} N_{,r}^1 & 0 & N_{,r}^2 & 0 & N_{,r}^3 & 0 & N_{,r}^4 & 0 \\ 0 & N_{,z}^1 & 0 & N_{,z}^2 & 0 & N_{,z}^3 & 0 & N_{,z}^4 \\ \frac{1}{r}N^1 & 0 & \frac{1}{r}N^2 & 0 & \frac{1}{r}N^3 & 0 & \frac{1}{r}N^4 & 0 \\ 0 & N_{,r}^1 & 0 & N_{,r}^2 & 0 & N_{,r}^3 & 0 & N_{,r}^4 \\ N_{,z}^1 & 0 & N_{,z}^2 & 0 & N_{,z}^3 & 0 & N_{,z}^4 & 0 \end{bmatrix} \begin{bmatrix} u_r^1 \\ u_z^1 \\ u_r^2 \\ u_z^2 \\ u_r^3 \\ u_z^3 \\ u_r^4 \\ u_z^4 \end{bmatrix} \rightarrow \left( \underline{\underline{L}}_u \right)_t = \underline{\underline{B}} \underline{\underline{u}}^e$$

$$\begin{bmatrix} N_{,r}^1 & N_{,z}^1 \\ N_{,r}^2 & N_{,z}^2 \\ N_{,r}^3 & N_{,z}^3 \\ N_{,r}^4 & N_{,z}^4 \end{bmatrix} = \begin{bmatrix} N_{b,1}^1 & N_{b,2}^1 \\ N_{b,1}^2 & N_{b,2}^2 \\ N_{b,1}^3 & N_{b,2}^3 \\ N_{b,1}^4 & N_{b,2}^4 \end{bmatrix} \begin{bmatrix} \xi_{1,r} & \xi_{1,z} \\ \xi_{2,r} & \xi_{2,z} \end{bmatrix} = \begin{bmatrix} N_{,1}^1 & N_{,2}^1 \\ N_{,1}^2 & N_{,2}^2 \\ N_{,1}^3 & N_{,2}^3 \\ N_{,1}^4 & N_{,2}^4 \end{bmatrix} \underline{\underline{J}}^{-T}$$

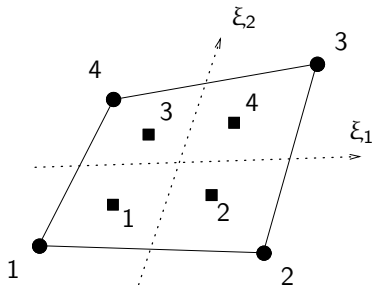
$$\underline{\underline{J}} = \begin{bmatrix} r_{,1} & z_{,1} \\ r_{,2} & z_{,2} \end{bmatrix} = \begin{bmatrix} N_{,1}^1 & N_{,2}^1 & N_{,1}^3 & N_{,2}^3 \\ N_{,1}^2 & N_{,2}^2 & N_{,1}^4 & N_{,2}^4 \end{bmatrix} \begin{bmatrix} r_e^1 & z_e^1 \\ r_e^2 & z_e^2 \\ r_e^3 & z_e^3 \\ r_e^4 & z_e^4 \end{bmatrix}$$

# Deformation matrix

$$\underline{F} = \begin{bmatrix} \frac{\partial r}{\partial r_0} & \frac{\partial r}{\partial z_0} & 0 \\ \frac{\partial z}{\partial r_0} & \frac{\partial z}{\partial z_0} & 0 \\ 0 & 0 & \frac{r}{r_0} \end{bmatrix}$$

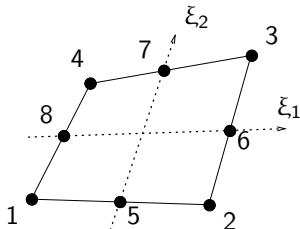
$$\begin{aligned} \begin{bmatrix} \frac{\partial r}{\partial r_0} & \frac{\partial z}{\partial r_0} \\ \frac{\partial r}{\partial z_0} & \frac{\partial z}{\partial z_0} \end{bmatrix} &= \begin{bmatrix} N_{,r0}^1 & N_{,r0}^2 & N_{,r0}^3 & N_{,r0}^4 \\ N_{,z0}^1 & N_{,z0}^2 & N_{,z0}^3 & N_{,z0}^4 \end{bmatrix} \begin{bmatrix} r_e^1 & z_e^1 \\ r_e^2 & z_e^2 \\ r_e^3 & z_e^3 \\ r_e^4 & z_e^4 \end{bmatrix} \\ &= \begin{bmatrix} \xi_{1,r0} & \xi_{2,r0} \\ \xi_{1,z0} & \xi_{2,z0} \end{bmatrix} \begin{bmatrix} N_{,2}^1 & N_{,2}^2 & N_{,2}^3 & N_{,2}^4 \\ N_{,2}^1 & N_{,2}^2 & N_{,2}^3 & N_{,2}^4 \end{bmatrix} \begin{bmatrix} r_e^1 & z_e^1 \\ r_e^2 & z_e^2 \\ r_e^3 & z_e^3 \\ r_e^4 & z_e^4 \end{bmatrix} \\ &= \underline{J}_0^{-1} \underline{J} \end{aligned}$$

# Numerical integration



$ip$	$\xi_1$	$\xi_2$	$\zeta$
1	$-\frac{1}{3}\sqrt{3}$	$-\frac{1}{3}\sqrt{3}$	1
2	$\frac{1}{3}\sqrt{3}$	$-\frac{1}{3}\sqrt{3}$	1
3	$-\frac{1}{3}\sqrt{3}$	$\frac{1}{3}\sqrt{3}$	1
4	$\frac{1}{3}\sqrt{3}$	$\frac{1}{3}\sqrt{3}$	1

# Eight-node quadrilateral element



interpolation functions quadratic in  $(\xi_1, \xi_2)$

$$N^1 = \frac{1}{4}(\xi_1 - 1)(\xi_2 - 1)(-\xi_1 - \xi_2 - 1)$$

$$N^2 = \frac{1}{4}(\xi_1 + 1)(\xi_2 - 1)(-\xi_1 + \xi_2 + 1)$$

$$N^3 = \frac{1}{4}(\xi_1 + 1)(\xi_2 + 1)(\xi_1 + \xi_2 - 1)$$

$$N^4 = \frac{1}{4}(\xi_1 - 1)(\xi_2 + 1)(\xi_1 - \xi_2 + 1)$$

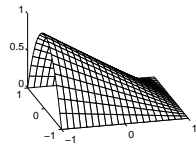
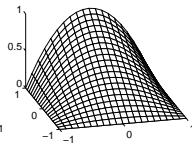
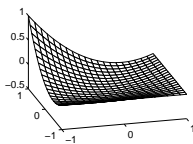
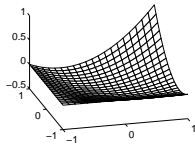
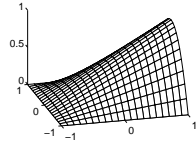
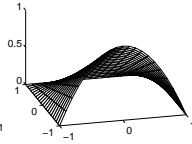
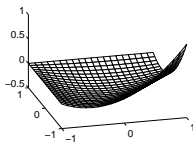
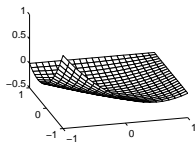
$$N^5 = \frac{1}{2}(\xi_1^2 - 1)(\xi_2 - 1)$$

$$N^6 = \frac{1}{2}(-\xi_1 - 1)(\xi_2^2 - 1)$$

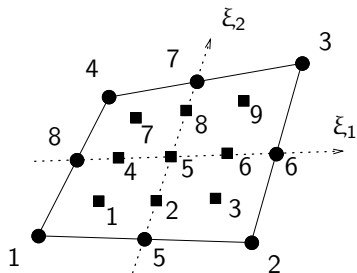
$$N^7 = \frac{1}{2}(\xi_1^2 - 1)(-\xi_2 - 1)$$

$$N^8 = \frac{1}{2}(\xi_1 - 1)(\xi_2^2 - 1)$$

# Shape functions



# Numerical integration



$$a = 0.77459; \quad p = 0.55556; \quad q = 0.88889$$

$ip$	$\xi_1$	$\xi_2$	$\zeta$
1	$-a$	$-a$	$p \times p$
2	0	$-a$	$p \times q$
3	$a$	$-a$	$p \times p$
4	$-a$	0	$p \times q$
5	0	0	$q \times q$
6	$a$	0	$p \times q$
7	$-a$	$a$	$p \times p$
8	0	$a$	$p \times q$
9	$a$	$a$	$p \times p$