

1) For rectangular / cartesian coordinate system:-

Let (x_0, y_0, z_0) is the centre of the cv. & c measure the properties

$$\vec{V} = U\hat{i} + V\hat{j} + W\hat{k}$$

$$A = Ax\hat{i} + Ay\hat{j} + Az\hat{k}$$

$$\rho(x_0, y_0, z_0)$$

\therefore we know the prop. at centre

\therefore we are finding prop. at centre of face ABCD suppose.

$\therefore \rho_{x+\frac{dx}{2}} = \rho$ denotes at a dist $\frac{dx}{2}$ from origin (keeping consideration only in x -direction due to variation constant).

\therefore by Taylor expansion :-

$$\rho_{x+\frac{dx}{2}} = \rho + \left(\frac{1}{1!}\right) \left(\frac{\partial \rho}{\partial x}\right) \left(\frac{dx}{2}\right) + \left(\frac{1}{2!}\right) \left(\frac{\partial^2 \rho}{\partial x^2}\right) \left(\frac{dx}{2}\right)^2 + \dots$$

neglecting higher order terms.

$$\therefore \rho_{x+\frac{dx}{2}} \approx \rho + \left(\frac{\partial \rho}{\partial x}\right) \frac{dx}{2} \quad \text{--- (1)}$$

\therefore for velocity in x -direction :-

$$U_{x+\frac{dx}{2}} = U + \left(\frac{\partial U}{\partial x}\right) \frac{dx}{2} \quad \text{--- (2)}$$

\therefore properties at centre of face ABCD suppose.

By Taylor expansion:-

$$\rho_{x-\frac{dx}{2}} \approx \rho - \left(\frac{\partial \rho}{\partial x}\right) \frac{dx}{2} \quad \text{--- (3)}$$

\therefore for velocity in x -direction :-

$$U_{x-\frac{dx}{2}} = U - \left(\frac{\partial U}{\partial x}\right) \frac{dx}{2} \quad \text{--- (4)}$$

\therefore net mass flow rate out along x -direction

$$\dot{m}_{out} = (\text{Volume out} - \text{Volume in}) \cdot \frac{\text{mass flow rate}}{x}$$

$$\begin{aligned}
 &= \left(P_{\frac{x+d}{2}} \cdot U_{\frac{x+d}{2}} - P_{\frac{x-d}{2}} \cdot U_{\frac{x-d}{2}} \right) dy dz \quad \text{from } \textcircled{1}, \textcircled{2}, \textcircled{3}, \textcircled{4} \\
 &= \left[\left(P + \frac{\partial P}{\partial x} \cdot \frac{dx}{2} \right) \left(U + \frac{\partial U}{\partial x} \cdot \frac{dx}{2} \right) - \left(P - \frac{\partial P}{\partial x} \cdot \frac{dx}{2} \right) \left(U - \frac{\partial U}{\partial x} \cdot \frac{dx}{2} \right) \right] dy dz \\
 &= \left[P_U + P \cdot \frac{\partial U}{\partial x} \cdot \frac{dx}{2} + U \frac{\partial P}{\partial x} \cdot \frac{dx}{2} - PU + P \cdot \frac{\partial U}{\partial x} \cdot \frac{dx}{2} + U \cdot \frac{\partial P}{\partial x} \cdot \frac{dx}{2} \right] dy dz \\
 &= \left[P \frac{\partial U}{\partial x} \cdot dx + U \cdot \frac{\partial P}{\partial x} \cdot dx \right] dy dz \quad (\text{neglecting prod. of derivatives}) \\
 &= \left[\rho \frac{dU}{dx} + U \frac{\partial P}{\partial x} \right] dx dy dz.
 \end{aligned}$$

$$= \frac{\partial (\rho U)}{\partial x} \quad \text{--- } \textcircled{I}$$

∴ similarly for face CDHG & ABFE (z-direction)

net mass flow rate out along -z-direction

$$= (\text{Volume out} - \text{Volume in}) z$$

$$= \frac{\partial (\rho w)}{\partial z} \quad \text{--- } \textcircled{II}$$

∴ similarly for face AEDH & BFGC (y-direction)

net mass flow rate out along -y-direction.

$$= (\text{Volume out} - \text{Volume in}) y$$

$$= \frac{\partial (\rho V)}{\partial y} \quad \text{--- } \textcircled{III}$$

∴ from \textcircled{I} , \textcircled{II} , \textcircled{III}

through C.S.

net mass flow rate out = Net sum (mass flow rate out)
along (x, y, z) CV direction or N out through C.S.

∴ Time rate of change of mass within CV

$$\text{within } C.V. = \frac{\partial}{\partial t} (\rho \cdot dV) = \frac{\partial \rho}{\partial t} \cdot dV \quad \text{generate C.S.}$$

∴ As per law of conservation of mass:- From RTT, $\frac{dm}{dt} = \frac{\partial}{\partial t} \int_C V \rho dV + \int_S \rho V dA$

$$\left[\frac{\partial}{\partial x} (\rho U) + \frac{\partial}{\partial y} (\rho V) + \frac{\partial}{\partial z} (\rho W) \right] dV \textcircled{IV} + \frac{\partial \rho}{\partial t} \cdot dV = \frac{dm}{dt}$$

$$\left[\frac{\partial}{\partial x} (\rho U) + \frac{\partial}{\partial y} (\rho V) + \frac{\partial}{\partial z} (\rho W) + \frac{\partial \rho}{\partial t} \right] dV = 0 \quad [E: \frac{dm}{dt} = 0 \text{ in C.M.}]$$

$$\frac{\partial P}{\partial t} + \frac{\partial (PV)}{\partial x} + \frac{\partial (PV)}{\partial y} + \frac{\partial (PV)}{\partial z} = 0 \quad \text{as } \frac{\partial V}{\partial x} = \text{constant}$$

$$\boxed{\frac{\partial P}{\partial t} + \vec{V} \cdot \vec{\nabla} P = 0} \quad \text{where } \vec{V} = u\hat{i} + v\hat{j} + w\hat{k}, \quad \vec{\nabla} = i\frac{\partial}{\partial x} + j\frac{\partial}{\partial y} + k\frac{\partial}{\partial z}$$

$$\therefore \frac{\partial P}{\partial t} + P \vec{\nabla} \cdot \vec{V} + (\vec{V} \cdot \vec{\nabla}) P = 0$$

$$\therefore \left[\frac{\partial}{\partial t} + \vec{V} \cdot \vec{\nabla} \right] P + P \vec{\nabla} \cdot \vec{V} = 0$$

$$\therefore \boxed{\frac{DP}{Dt} + P \vec{\nabla} \cdot \vec{V} = 0} \quad \text{--- (B)}$$

$$\text{Here } \frac{D}{Dt} = \frac{\partial}{\partial t} + (\vec{V} \cdot \vec{\nabla})$$

material local. \downarrow
 derivative derivative convective
 derivative derivative part
 \downarrow
 unsteady
 part

$\frac{\partial P}{\partial t}$ = look your eyes ~~at a~~ at a certain pt. & observe the characteristics ~~change in~~ ~~density (P)~~ with time.
 $\frac{DP}{Dt}$ = observes a property at a particular pt
 = fix your eyes at a fluid particle and observe the change in density.

Hence.

$$\text{Rate of change of mass. within a cv} + \text{Net mass flow rate out through a c-s.} = 0$$

$\{ \text{ex spd} = \text{intensity} \}$

$\{ \text{volume at each face is constant at the } x, y, z \text{ coordinates are} \}$
 $\{ \text{not changing their values with that direction} \}$

$\{ \text{for uncompressible flow: - (not changing with space) } \}$

$$\frac{D}{Dt} = \frac{\partial}{\partial t} + 0 \quad \frac{DP}{Dt} = \left(\frac{\partial}{\partial t} + u\frac{\partial}{\partial x} + v\frac{\partial}{\partial y} \right) P = 0$$

\rightarrow for steady flow: - (not changing with time): - $\frac{DP}{Dt} = 0$

$$\frac{DP}{Dt} + P \cdot \vec{\nabla} \cdot \vec{V} = 0$$

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} + \frac{\partial w}{\partial z} = 0$$

For steady flow: - From eqn A $\Rightarrow \frac{DP}{Dt} + \vec{V} \cdot \vec{\nabla} P = 0 \therefore \vec{\nabla} \cdot \vec{P} \vec{V} = 0$

2) Equation of continuity in cylindrical coordinate syst.

(r, θ, z) :-

Here r, θ are varying

and $z = \text{constant}$.

Hence volume is $\pi r^2 z$

but vary in case of

direction in r .

We assume the

following properties at $\theta = 0$

the surface element

considered :-

$$\vec{V} = V_r e_r + V_\theta e_\theta + V_z e_z$$

$$P(r, \theta, z)$$

$$\vec{A} = A_r e_r + A_\theta e_\theta + A_z e_z$$

- Considering each surface :-

∴ Along the r -direction; net mass flow rate :-

$$= (\text{Intensity out} - \text{Intensity in}) r \times \text{vol. (in general)}.$$

$$= (P_{\text{out}} \cdot V_{\text{out}} \cdot A_{\text{out}} - P_{\text{in}} \cdot V_{\text{in}} \cdot A_{\text{in}})$$

$$= P_{r+\frac{dr}{2}} \cdot V_{r+\frac{dr}{2}} \cdot (r+\frac{dr}{2}) d\theta dz - P_{r-\frac{dr}{2}} \cdot V_{r-\frac{dr}{2}} \cdot (r-\frac{dr}{2}) d\theta dz.$$

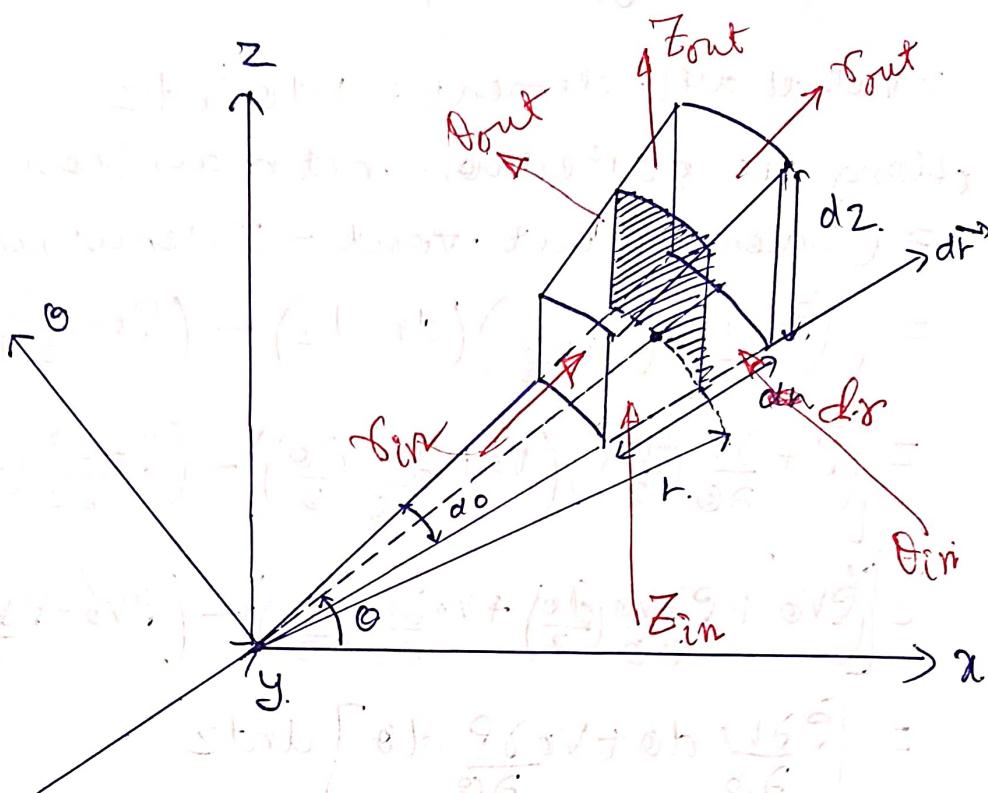
$$= \left[\left(P + \frac{\partial P}{\partial r} \left(\frac{dr}{2} \right) \right) \left(V + \frac{\partial V}{\partial r} \left(\frac{dr}{2} \right) \right) \left(r + \frac{dr}{2} \right) - \left(P - \frac{\partial P}{\partial r} \left(\frac{dr}{2} \right) \right) \left(V - \frac{\partial V}{\partial r} \left(\frac{dr}{2} \right) \right) \left(r - \frac{dr}{2} \right) \right] d\theta dz$$

$$= \left[\left(V_r + \frac{\partial V_r}{\partial r} \left(\frac{dr}{2} \right) + V_r \frac{\partial P}{\partial r} \left(\frac{dr}{2} \right) \right) \left(r + \frac{dr}{2} \right) - \left(V_r - \frac{\partial V_r}{\partial r} \left(\frac{dr}{2} \right) - V_r \frac{\partial P}{\partial r} \left(\frac{dr}{2} \right) \right) \left(r - \frac{dr}{2} \right) \right] d\theta dz$$

$$= \left[P V_r dr + \frac{\partial V_r}{\partial r} \left(\frac{dr}{2} \right) (2r) + V_r \frac{\partial P}{\partial r} \left(\frac{dr}{2} \right) (2r) \right] d\theta dz$$

$$= \left[P V_r + \frac{\partial V_r}{\partial r} \cdot r + V_r \frac{\partial P}{\partial r} \cdot r \right] dr d\theta dz$$

$$= \left[P V_r + P \left(\frac{\partial V_r}{\partial r} + V_r \frac{\partial P}{\partial r} \right) \right] dr d\theta dz$$



$$= \left(\rho v_r + r \frac{\partial}{\partial r} (\rho v_r) \right) dr d\theta dz \quad \text{--- (I)}$$

$$\approx \text{vol. of diff. element} = r d\theta dr dz = dt$$

along the r direction, net mass flow rate =

$$= (\text{Intensity out} \cdot A_{\text{out}} - \text{Intensity in} \cdot A_{\text{in}}) \cdot (r \cdot d\theta \cdot dz)$$

$$= \left(\left(\rho_0 + \frac{d\rho}{d\theta} \left(\frac{d\theta}{2} \right) \right) (v_0 + \frac{\partial v_0}{\partial \theta} \left(\frac{d\theta}{2} \right)) - \left(\rho_0 - \frac{d\rho}{d\theta} \left(\frac{d\theta}{2} \right) \right) (v_0 - \frac{\partial v_0}{\partial \theta} \left(\frac{d\theta}{2} \right)) \right) dr \cdot dz$$

$$= \left[\left(\rho + \frac{\partial \rho}{\partial \theta} \left(\frac{d\theta}{2} \right) \right) \left(v_0 + \frac{\partial v_0}{\partial \theta} \left(\frac{d\theta}{2} \right) \right) - \left(\rho - \frac{\partial \rho}{\partial \theta} \left(\frac{d\theta}{2} \right) \right) \left(v_0 - \frac{\partial v_0}{\partial \theta} \left(\frac{d\theta}{2} \right) \right) \right] dr dz$$

$$= \left[\left(\rho v_0 + \rho \frac{\partial v_0}{\partial \theta} \left(\frac{d\theta}{2} \right) + v_0 \frac{\partial \rho}{\partial \theta} \left(\frac{d\theta}{2} \right) \right) - \left(\rho v_0 - \rho \frac{\partial v_0}{\partial \theta} \left(\frac{d\theta}{2} \right) - v_0 \frac{\partial \rho}{\partial \theta} \left(\frac{d\theta}{2} \right) \right) \right] dr dz$$

$$= \left[\rho \frac{\partial v_0}{\partial \theta} d\theta + v_0 \frac{\partial \rho}{\partial \theta} d\theta \right] dr dz$$

$$= \frac{\partial}{\partial \theta} (\rho v_0) dr d\theta dz \quad \text{--- (II)}$$

along the z direction, net mass flow rate =

$$= (\text{Intensity out} \cdot A_{\text{out}} - \text{Intensity in} \cdot A_{\text{in}}) \cdot dz$$

$$= \left[\left(\rho_z + \frac{d\rho}{dz} \left(\frac{dz}{2} \right) \right) (v_z + \frac{\partial v_z}{\partial z} \left(\frac{dz}{2} \right)) - \left(\rho_z - \frac{d\rho}{dz} \left(\frac{dz}{2} \right) \right) (v_z - \frac{\partial v_z}{\partial z} \left(\frac{dz}{2} \right)) \right] r dr d\theta dz$$

$$= \left[\left(\rho + \frac{\partial \rho}{\partial z} \left(\frac{dz}{2} \right) \right) \left(v_z + \frac{\partial v_z}{\partial z} \left(\frac{dz}{2} \right) \right) - \left(\rho - \frac{\partial \rho}{\partial z} \left(\frac{dz}{2} \right) \right) \left(v_z - \frac{\partial v_z}{\partial z} \left(\frac{dz}{2} \right) \right) \right] r dr d\theta dz$$

$$= \left[\left(\rho v_z + \rho \frac{\partial v_z}{\partial z} \left(\frac{dz}{2} \right) + v_z \frac{\partial \rho}{\partial z} \left(\frac{dz}{2} \right) \right) - \left(\rho v_z - \rho \frac{\partial v_z}{\partial z} \left(\frac{dz}{2} \right) - v_z \frac{\partial \rho}{\partial z} \left(\frac{dz}{2} \right) \right) \right] r dr d\theta dz$$

$$= \left[\rho \frac{\partial v_z}{\partial z} + v_z \frac{\partial \rho}{\partial z} \right] r dr d\theta dz$$

$$= r \frac{\partial}{\partial z} (\rho v_z) dr d\theta dz \quad \text{--- (III)}$$

from (IV) (V) (VI), net mass flow through C.S.

$$= \left[\left(\rho V_r + r \frac{\partial}{\partial r} (\rho V_r) \right) dr d\theta dz \right] + \left[\frac{\partial}{\partial \theta} (\rho V_\theta) dr d\theta dz \right] + \left[r \frac{\partial}{\partial z} (\rho V_z) dr d\theta dz \right]$$

$$= \left[\rho V_r + r \frac{\partial}{\partial r} (\rho V_r) + \frac{\partial}{\partial \theta} (\rho V_\theta) + r \frac{\partial}{\partial z} (\rho V_z) \right] dr d\theta dz$$

$$= \left[\frac{\partial}{\partial r} (r \cdot \rho V_r) + \frac{\partial}{\partial \theta} (\rho V_\theta) + r \frac{\partial}{\partial z} (\rho V_z) \right] dr d\theta dz \quad \text{--- (IV)}$$

\therefore Time rate of change of mass within CV = $\frac{\partial}{\partial t} (\rho \cdot r d\theta dr dz) = \frac{\partial \rho}{\partial t} \cdot r d\theta dr dz$ $\quad \text{--- (V)}$

Hence continuity in r-θ-z is :-

$$= (IV) + (V) = 0$$

$$= \left[\frac{\partial}{\partial r} (r \cdot \rho V_r) + \frac{\partial}{\partial \theta} (\rho V_\theta) + r \frac{\partial}{\partial z} (\rho V_z) \right] + \left(r \frac{\partial \rho}{\partial t} \right) dr d\theta dz = 0$$

ans:

$$\therefore \left[\frac{\partial \rho}{\partial t} + \left[1 \cdot \frac{\partial}{\partial r} (r \cdot \rho V_r) + \frac{1}{r} \cdot \frac{\partial}{\partial \theta} (\rho V_\theta) + \frac{\partial}{\partial z} (\rho V_z) \right] \right] = 0$$

$$\rightarrow \frac{DP}{Dt} = \frac{\partial \rho}{\partial t} + \vec{\nabla} \cdot (\rho \vec{V}) = 0 \quad \rightarrow \text{in general.}$$

\therefore for r, θ, z system:-

$$\vec{\nabla} = \hat{e}_r \frac{\partial}{\partial r} + \frac{1}{r} \hat{e}_\theta \frac{\partial}{\partial \theta} + \hat{e}_z \frac{\partial}{\partial z}$$

$$\vec{V} = V_r \hat{e}_r + V_\theta \hat{e}_\theta + V_z \hat{e}_z$$



for x-y-z system:-

$$\vec{\nabla} = \hat{i} \frac{\partial}{\partial x} + \hat{j} \frac{\partial}{\partial y} + \hat{k} \frac{\partial}{\partial z}$$

$$\vec{V} = U \hat{i} + V \hat{j} + W \hat{k}$$

→ mass flow through the differential area:-

$dm = \rho U ds$

density velocity over section area.

1) eq. of continuity in $x-y-z$ coordinates:-

$$\frac{DP}{Dt} = \frac{\partial P}{\partial t} + \left(\frac{\partial (\rho U)}{\partial x} + \frac{\partial (\rho V)}{\partial y} + \frac{\partial (\rho W)}{\partial z} \right)$$

∴ for $(x, y, z) \rightarrow (r \cos \theta, r \sin \theta, z)$

$$r = (x^2 + y^2)^{1/2}; \theta = \tan^{-1}(y/x)$$

$$2) \frac{DP}{Dt} = \frac{\partial P}{\partial t} + \left(\frac{1}{r} \frac{\partial (\rho r U_r)}{\partial r} + \frac{1}{r \sin \theta} \frac{\partial (\rho r v_\theta)}{\partial \theta} + \frac{\partial (\rho v_z)}{\partial z} \right)$$

i.e. eq. of continuity in $r-\theta-z$ coordinates

∴ for $(x, y, z) \rightarrow (r \sin \theta \sin \phi, r \sin \theta \cos \phi, r \cos \theta)$

$$r = (x^2 + y^2 + z^2)^{1/2}; \theta = \tan^{-1} \frac{(x^2 + y^2)^{1/2}}{z}; \phi = \tan^{-1} \left(\frac{y}{x} \right)$$

$$3) \frac{DP}{Dt} = \frac{\partial P}{\partial t} + \left(\frac{1}{r^2} \frac{\partial (r^2 \rho U_r)}{\partial r} + \frac{1}{r \sin \theta} \frac{\partial (\rho r v_\theta \sin \theta)}{\partial \theta} + \frac{1}{r \sin \theta} \frac{\partial (\rho v_\phi)}{\partial \phi} \right)$$

i.e. eq. of continuity in $r-\theta-\phi$ coordinates

$$\frac{639}{56} + \frac{6091}{864} + \frac{6}{56} = \nabla$$

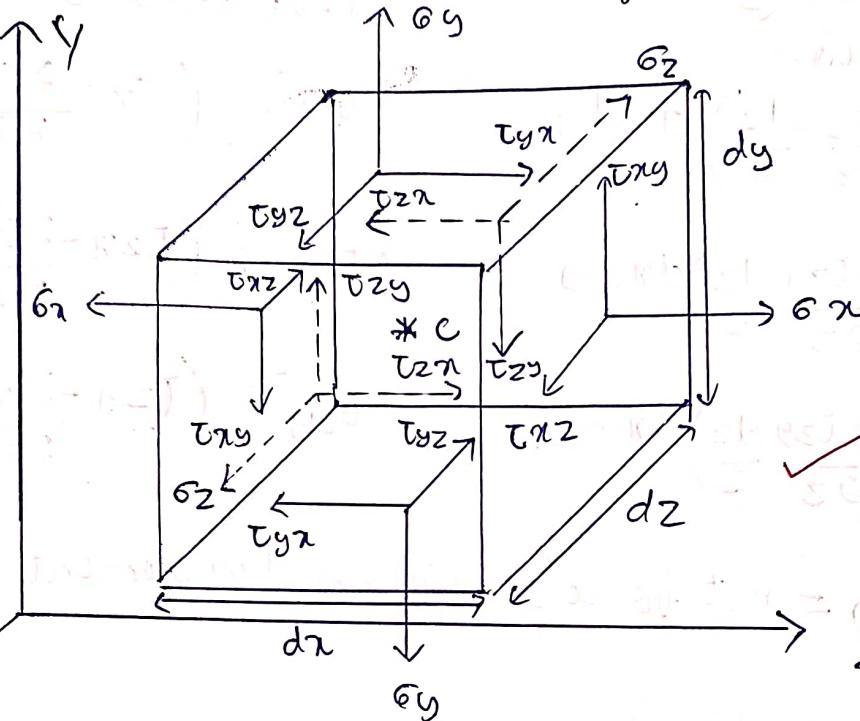
$$\frac{595V}{56} + \frac{699W}{864} + \frac{79W}{56} = \nabla$$

$$\frac{656R}{56} + \frac{686G}{864} + \frac{6}{56} = \nabla$$

$$R\bar{U} + G\bar{V} + B\bar{W} = \nabla$$

→ Navier-Stokes Equation :-
It is a Partial differential equation that describes the flow of incompressible fluids & frictionless fluids.

The elemental cube is a controlled volume element and experiences body forces & surface forces.



at pt c, the stresses present at $\frac{dx}{2}, \frac{dy}{2}, \frac{dz}{2}$ are
 $\sigma_x, \sigma_y, \sigma_z, \tau_{xy}, \tau_{xz}, \tau_{yz}, \tau_{zy}$

∴ for x plane:-

i) for x direction :- — (I)

$$F_{Gx}^+ = \left(\sigma_x + \frac{\partial \sigma_x}{\partial x} \cdot \frac{dx}{2} \right) dy dz$$

$$F_{Txxy}^+ = \left(\tau_{xy} + \frac{\partial \tau_{xy}}{\partial x} \cdot \frac{dx}{2} \right) dy dz$$

$$F_{Txz}^+ = \left(\tau_{xz} + \frac{\partial \tau_{xz}}{\partial x} \cdot \frac{dx}{2} \right) dy dz$$

for y plane:- — (II)

i) for y-direction

$$F_{Gy}^+ = \left(\tau_{yx} + \frac{\partial \tau_{yx}}{\partial y} \cdot \frac{dy}{2} \right) dx dz$$

$$F_{Tyx}^+ = \left(\tau_{xy} - \frac{\partial \tau_{xy}}{\partial y} \cdot \frac{dy}{2} \right) dx dz$$

ii) for y-direction:-

$$F_{Gy}^- = \left(\sigma_y - \frac{\partial \sigma_y}{\partial y} \cdot \frac{dy}{2} \right) dx dz$$

$$F_{Tyx}^- = \left(\tau_{xy} - \frac{\partial \tau_{xy}}{\partial y} \cdot \frac{dy}{2} \right) dx dz$$

$$F_{Txz}^- = \left(\tau_{xz} - \frac{\partial \tau_{xz}}{\partial y} \cdot \frac{dy}{2} \right) dx dz$$

ii) for z-direction

$$F_{Gz}^+ = \left(\tau_{yz} + \frac{\partial \tau_{yz}}{\partial z} \cdot \frac{dz}{2} \right) dx dy$$

$$F_{Tyz}^+ = \left(\tau_{yz} - \frac{\partial \tau_{yz}}{\partial z} \cdot \frac{dz}{2} \right) dx dy$$

$$F_{Gy}^+ = \left(\sigma_y + \frac{\partial \sigma_y}{\partial y} \cdot \frac{dy}{2} \right) dx dz$$

$$F_{Gy}^- = \left(\sigma_y - \frac{\partial \sigma_y}{\partial y} \cdot \frac{dy}{2} \right) dx dz$$

for z-plane:- \textcircled{III}

ii) in z-direction

i) +ve z-direction

$$F_{Gz}^+ = \left(\sigma_z + \frac{\partial \sigma_z}{\partial z} \cdot \frac{dz}{2} \right) dx dy$$

$$F_{Gz}^- = \left(\sigma_z - \frac{\partial \sigma_z}{\partial z} \cdot \frac{dz}{2} \right) dx dy$$

$$F_{Tzx}^+ = \left(\tau_{zx} + \frac{\partial \tau_{zx}}{\partial z} \cdot \frac{dz}{2} \right) dx dy$$

$$F_{Tzx}^- = \left(\tau_{zx} - \frac{\partial \tau_{zx}}{\partial z} \cdot \frac{dz}{2} \right) dx dy$$

$$F_{Tzy}^+ = \left(\tau_{zy} + \frac{\partial \tau_{zy}}{\partial z} \cdot \frac{dz}{2} \right) dx dy$$

$$F_{Tzy}^- = \left(\tau_{zy} - \frac{\partial \tau_{zy}}{\partial z} \cdot \frac{dz}{2} \right) dx dy$$

$\therefore d\vec{F} = \frac{d\vec{V}}{dt} \cdot dm = \text{net force acting on } dm \text{ element}$

$$d\vec{F} = \left[\begin{array}{l} d\vec{F}_s \\ (\text{surface}) \end{array} + \begin{array}{l} d\vec{F}_b \\ (\text{body}) \end{array} \right]_{xyz}$$

$$\therefore d\vec{F} = (dF_{sx} + dF_{bx})\hat{i} + (dF_{sy} + dF_{by})\hat{j} + (dF_{sz} + dF_{bz})\hat{k}$$

\therefore along x-direction

$$dF_{sx} = F_{Gx}^+ + F_{Gx}^- + F_{Tyx}^+ + F_{Tyx}^- + F_{Tzx}^+ + F_{Tzx}^- \quad (\text{from } \textcircled{I}, \textcircled{II}, \textcircled{III})$$

$$= \left[\left(\sigma_x + \frac{\partial \sigma_x}{\partial x} \cdot \frac{dx}{2} \right) - \left(\sigma_x - \frac{\partial \sigma_x}{\partial x} \cdot \frac{dx}{2} \right) \right] dy dz +$$

$$\left[\left(\tau_{yx} + \frac{\partial \tau_{yx}}{\partial y} \cdot \frac{dy}{2} \right) - \left(\tau_{yx} - \frac{\partial \tau_{yx}}{\partial y} \cdot \frac{dy}{2} \right) \right] dx dz +$$

$$\left[\left(\tau_{xz} + \frac{\partial \tau_{xz}}{\partial z} \cdot \frac{dz}{2} \right) - \left(\tau_{xz} - \frac{\partial \tau_{xz}}{\partial z} \cdot \frac{dz}{2} \right) \right] dx dy$$

$$dF_{sx} = \left[\frac{\partial \sigma_x}{\partial x} + \frac{\partial \tau_{yx}}{\partial y} + \frac{\partial \tau_{xz}}{\partial z} \right] dx dy dz \quad \boxed{1}$$

$$\text{also: } (dF_{bx}) = (\rho \cdot dt) g$$

along x -direction:-

$$dF_{xy} = F_{xy}^+ + F_{xy}^- + F_{txy}^+ + F_{txy}^- + F_{tyz}^+ + F_{tyz}^- \quad (\text{from } ① ② ③)$$

$$= \left[\left(G_y + \frac{\partial G_y}{\partial y} \frac{dy}{2} \right) - \left(G_y - \frac{\partial G_y}{\partial y} \frac{dy}{2} \right) \right] dxdz +$$

$$\left[\left(T_{xy} + \frac{\partial T_{xy}}{\partial x} \frac{dx}{2} \right) - \left(T_{xy} - \frac{\partial T_{xy}}{\partial x} \frac{dx}{2} \right) \right] dydz +$$

$$\left[\left(T_{zy} + \frac{\partial T_{zy}}{\partial z} \frac{dz}{2} \right) - \left(T_{zy} - \frac{\partial T_{zy}}{\partial z} \frac{dz}{2} \right) \right] dx dy$$

$$dF_{xy} = \left[\frac{\partial G_y}{\partial y} \cdot 0 + \frac{\partial T_{xy}}{\partial x} + \frac{\partial T_{zy}}{\partial z} \right] dxdydz \quad \boxed{2}$$

$$\text{also } dF_{bx} = (edt)G_y$$

along y -direction:-

$$dF_{yz} = F_{yz}^+ + F_{yz}^- + F_{txz}^+ + F_{txz}^- + F_{oyz}^+ + F_{oyz}^- \quad (\text{from } ④ ⑤ ⑥)$$

$$= \left[\left(G_z + \frac{\partial G_z}{\partial z} \frac{dz}{2} \right) - \left(G_z - \frac{\partial G_z}{\partial z} \frac{dz}{2} \right) \right] dx dy +$$

$$\left[\left(T_{xz} + \frac{\partial T_{xz}}{\partial x} \frac{dx}{2} \right) - \left(T_{xz} - \frac{\partial T_{xz}}{\partial x} \frac{dx}{2} \right) \right] dy dz +$$

$$\left[\left(T_{yz} + \frac{\partial T_{yz}}{\partial y} \frac{dy}{2} \right) - \left(T_{yz} - \frac{\partial T_{yz}}{\partial y} \frac{dy}{2} \right) \right] dx dz$$

$$dF_{yz} = \left[\frac{\partial G_z}{\partial z} + \frac{\partial T_{xz}}{\partial x} + \frac{\partial T_{yz}}{\partial y} \right] dx dy dz \quad \boxed{3}$$

$$\text{also } dF_{bz} = (edt)G_z$$

$$\therefore d\vec{F} = \left(\frac{edt + Du}{Dv} \right) \hat{i} + \left(edt + \frac{Dv}{Dw} \right) \hat{j} + \left(edt + \frac{Dw}{Dv} \right) \hat{k} \quad \boxed{4}$$

comparing in each direction:-

from ④ & ①:-

$$\rho \frac{d\cancel{\frac{Dv}{Dt}}}{dt} = dF_{sx} + dF_{bx} \quad (\because dndydz = dt) \\ = \left(\frac{\partial G_x}{\partial n} + \frac{\partial T_yx}{\partial n} + \frac{\partial T_zx}{\partial z} \right) dndydz + \rho \cancel{\frac{Dg_x}{Dt}}$$

$$\therefore \frac{Dv}{Dt} = \frac{1}{\rho} \cdot \frac{\partial G_x}{\partial n} + \frac{1}{\rho} \frac{\partial T_yx}{\partial n} + \frac{1}{\rho} \frac{\partial T_zx}{\partial z} + g_x \quad \text{--- } A$$

from ④ & ②:-

$$\rho \frac{d\cancel{\frac{Dv}{Dt}}}{dt} = dF_{sy} + dF_{by} \quad (\because dndydz = dt) \\ = \left(\frac{\partial G_y}{\partial y} + \frac{\partial T_ay}{\partial n} + \frac{\partial T_zy}{\partial z} \right) dndydz + \rho \cancel{\frac{Dgy}{Dt}} \\ \therefore \frac{Dv}{Dt} = \frac{1}{\rho} \frac{\partial G_y}{\partial y} + \frac{1}{\rho} \frac{\partial T_ay}{\partial n} + \frac{1}{\rho} \frac{\partial T_zy}{\partial z} + g_y \quad \text{--- } B$$

from ⑤ & ③:-

$$\rho \frac{d\cancel{\frac{Dw}{Dt}}}{dt} = dF_{sz} + dF_{bz} \quad (\because dndydz = dt) \\ = \left(\frac{\partial G_z}{\partial z} + \frac{\partial T_ayz}{\partial n} + \frac{\partial T_zyz}{\partial y} \right) dndydz + \rho \cancel{\frac{Dgz}{Dt}} \\ \therefore \frac{Dw}{Dt} = \frac{1}{\rho} \frac{\partial G_z}{\partial z} + \frac{1}{\rho} \frac{\partial T_ayz}{\partial n} + \frac{1}{\rho} \frac{\partial T_zyz}{\partial y} + g_z \quad \text{--- } C$$

Ans. (we need to know proof) :-

$$T_{yx} = T_{xy} = M \left(\frac{\partial v}{\partial n} + \frac{\partial u}{\partial y} \right)$$

$$T_{zx} = T_{xz} = M \left(\frac{\partial w}{\partial n} + \frac{\partial u}{\partial z} \right)$$

$$T_{zy} = T_{yz} = M \left(\frac{\partial w}{\partial y} + \frac{\partial v}{\partial z} \right)$$

$$G_x = -P - \frac{2}{3} M \vec{J} \cdot \vec{V} + 2M \frac{\partial v}{\partial x}$$

$$G_y = -P - \frac{2}{3} M \vec{J} \cdot \vec{V} + 2M \frac{\partial v}{\partial y}$$

$$G_z = -P - \frac{2}{3} M \vec{J} \cdot \vec{V} + 2M \frac{\partial w}{\partial z}$$

Hydrostatic pressure, volume dilatation

here in β

p = thermodynamic pressure

(2/3) $\mu \vec{\nabla} \cdot \vec{v}$ = volume dilation

$2\mu \frac{\partial \text{Velocity in direction}}{\partial \text{direction}}$ = linear dilation

$\left. \begin{array}{l} (\mu = \text{absolute or} \\ \text{dynamic viscosity}) \end{array} \right\}$

from (2) & (3) in (A), (B), (C) :- $\left\{ \nu = \mu/\rho = \text{kinematic viscosity} \right\}$

$$\frac{\partial u}{\partial t} = \frac{1}{\rho} \frac{\partial}{\partial x} \left[-P - \frac{2}{3} \mu \vec{\nabla} \cdot \vec{v} + 2\mu \frac{\partial u}{\partial x} \right] + \frac{1}{\rho} \frac{\partial}{\partial y} \left[\left(\frac{\partial v}{\partial x} + \frac{\partial u}{\partial y} \right) \mu \right] + \frac{1}{\rho} \frac{\partial}{\partial z} \left[\mu \left(\frac{\partial w}{\partial x} + \frac{\partial v}{\partial z} \right) \right] + g_x \quad \text{(a)}$$

$$\frac{\partial v}{\partial t} = \frac{1}{\rho} \frac{\partial}{\partial y} \left[-P - \frac{2}{3} \mu \vec{\nabla} \cdot \vec{v} + 2\mu \frac{\partial v}{\partial y} \right] + \frac{1}{\rho} \frac{\partial}{\partial z} \left[\mu \left(\frac{\partial v}{\partial x} + \frac{\partial w}{\partial y} \right) \right] + g_y \quad \text{(b)}$$

$$\frac{\partial w}{\partial t} = \frac{1}{\rho} \frac{\partial}{\partial z} \left[-P - \frac{2}{3} \mu \vec{\nabla} \cdot \vec{v} + 2\mu \frac{\partial w}{\partial z} \right] + \frac{1}{\rho} \frac{\partial}{\partial y} \left[\mu \left(\frac{\partial w}{\partial x} + \frac{\partial v}{\partial z} \right) \right] + g_z \quad \text{(c)}$$

Considering incompressible fluid :- $\vec{\nabla} \cdot \vec{v} = 0$ (refer RTT)

$$\frac{\partial u}{\partial t} = g_x + \frac{1}{\rho} \frac{\partial(-P)}{\partial x} + \nu \left[\frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2} + \frac{\partial^2 u}{\partial z^2} \right] = g_x - \frac{\partial P}{\partial x} \frac{1}{\rho} + \nu \nabla^2 u \quad \text{(a.i)}$$

$$\frac{\partial v}{\partial t} = g_y + \frac{1}{\rho} \frac{\partial(-P)}{\partial y} + \nu \left[\frac{\partial^2 v}{\partial x^2} + \frac{\partial^2 v}{\partial y^2} + \frac{\partial^2 v}{\partial z^2} \right] = g_y - \frac{\partial P}{\partial y} \frac{1}{\rho} + \nu \nabla^2 v \quad \text{(b.i)}$$

$$\frac{\partial w}{\partial t} = g_z + \frac{1}{\rho} \frac{\partial(-P)}{\partial z} + \nu \left[\frac{\partial^2 w}{\partial x^2} + \frac{\partial^2 w}{\partial y^2} + \frac{\partial^2 w}{\partial z^2} \right] = g_z - \frac{\partial P}{\partial z} \frac{1}{\rho} + \nu \nabla^2 w \quad \text{(c.i)}$$

from (a) (b) (c) :-

$$\frac{D\vec{V}}{Dt} = \vec{g} - \frac{1}{\rho} \vec{\nabla} \cdot P + V \vec{V} \cdot \vec{\nabla}^2 \vec{V}$$

inertial force per unit volume

gravitational force.

pressure force.

viscous force per unit volume.

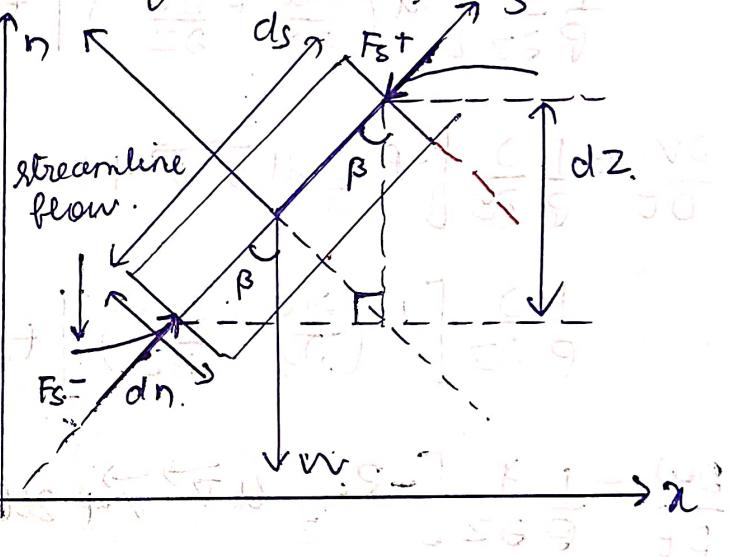
→ Euler's Equation & Bernoulli's Equation:-

Consider a streamlined flow as shown in figure with the coordinate axis n-s.

considering a streamline element, along the s-axis:-

$$F_s^+ = \left(P + \frac{\partial P}{\partial s} \cdot \frac{ds}{2} \right) dn \cdot d\pi \quad \text{--- (1)}$$

$$F_s^- = \left(P - \frac{\partial P}{\partial s} \cdot \frac{ds}{2} \right) dn \cdot d\pi \quad \text{--- (2)}$$



using Newton's 2nd law of motion :-

$$\vec{F}_{\text{net}} = M \cdot \vec{a} \quad \text{--- considering along s-axis}$$

$$F_s^- - F_s^+ - W \cos \beta = M a_s$$

$$\left[\left(P - \frac{\partial P}{\partial s} \cdot \frac{ds}{2} \right) - \left(P + \frac{\partial P}{\partial s} \cdot \frac{ds}{2} \right) \right] dn \cdot d\pi - (Pd + g) \cos \beta = (Pd + g) a_s$$

$$-\frac{\partial P}{\partial s} \cdot ds \cdot dn \cdot d\pi - (Pd + g) \frac{(dz)}{(ds)} = (Pd + g) a_s \quad (\because \cos \beta = dz/ds \quad \text{and } dn \cdot ds = dt)$$

$$-\frac{1}{\rho} \frac{\partial P}{\partial s} - g \frac{dz}{ds} = a_s = \frac{DV}{Dt} \Big|_s \quad \text{--- (1)}$$

$$\therefore \frac{DV}{Dt} = \frac{\partial V}{\partial t} + V_s \frac{dv}{ds} + V_n \frac{dv}{dn} \quad (\because \text{streamline element is flowing in s-direction only} \therefore V_n = 0)$$

$$\frac{DV}{Dt} \Big|_s = \frac{\partial V}{\partial t} + V_s \frac{dv}{ds} = \frac{\partial V}{\partial t} + V \frac{dv}{ds} \quad (\because \vec{V} = V_s \hat{s} + V_n \hat{n}) \quad \text{--- (2)}$$

from ① & ② :-

$$\left(-\frac{1}{\rho} \frac{\partial P}{\partial s} - g \frac{dz}{ds} = \frac{\partial V}{\partial t} + \frac{V dV}{ds} \right) \times ds$$

$$\therefore \frac{\partial V}{\partial t} \cdot ds + V dV + \frac{1}{\rho} \frac{\partial P}{\partial s} \cdot ds + g dz = 0 \quad \text{--- ③}$$

$$\approx V dV = d(V^2/2) \quad \text{--- ④}$$

$$\therefore \text{let } s = s(x_1, y_1, z)$$

$$\therefore \text{absolute derivative} = ds = \frac{\partial s}{\partial x_1} dx_1 + \frac{\partial s}{\partial y_1} dy_1 + \frac{\partial s}{\partial z} dz$$

$$\therefore \text{for } P = P(s)$$

$$\therefore \text{absolute derivative} = dP = \frac{\partial P}{\partial s} \cdot ds + \frac{\partial P}{\partial x_1} dx_1 + \frac{\partial P}{\partial y_1} dy_1 + \frac{\partial P}{\partial z} dz$$

$$\therefore \text{along } N=\text{const}, \frac{\partial P}{\partial x_1} = 0$$

$$\therefore dP = \frac{\partial P}{\partial s} \cdot ds \quad \text{--- ⑤}$$

from ③ ④ ⑤ :-

$$\frac{\partial V}{\partial t} \cdot ds + d\left(\frac{V^2}{2}\right) + \frac{dP}{\rho} + d(gz) = 0 \quad (\because g \text{ is a constant})$$

considering steady flow. $\therefore (\partial V/\partial t) = 0$

$$\therefore \boxed{\frac{dP}{\rho} + d\left(\frac{V^2}{2}\right) + d(gz) = 0} \rightarrow \text{Euler's equation}$$

This eq. holds true for the assumptions:-

1) only pressure & gravity forces are dominant & all other forces are negligible. i.e. inviscid fluid.

2) Flow is steady.

3) The fluid is ideal & non-viscous.

→ deriving Bernoulli's equation from Euler's equation :-
for an incompressible fluid, P is constant :-

∴ integrating Euler's equation throughout :-

$$\boxed{\frac{P}{\rho} + gz + \frac{V^2}{2} = \text{constant}} \quad \text{--- ①}$$

$$\frac{P}{\rho} + z + \frac{V^2}{2g} = \text{constant}$$

→ ②

$$\frac{P}{\rho} + z + \frac{V^2}{2g} = \frac{P_1}{\rho_1} + \frac{z_1}{g} + \frac{V_1^2}{2g}$$

① & ② are the different form of Bernoulli's equation or energy equation where last term in ② represents the energy per unit weight of the fluid.

Hence assumptions made Bernoulli's equation:-

- i) The fluid is incompressible
- ii) The flow is steady & continuous
- iii) The fluid is non-viscous
- iv) The flow is irrotational
- v) The gravity & pressure forces are only considered
- vi) The equation is applicable along a stream line only
- vii) The velocity is uniform over the cross-section

→ Limitations of Bernoulli's Equation :-

- i) The fluid is not ideal. Hence due to no-slip condition, the velocity at the fixed boundary is zero & it does not increase.
- ii) It means, the velocity across the section is not uniform
- iii) Practically, in addition to gravity & pressure forces, some forces like the viscous force is also involved in the motion of the fluid.
- iv) When the fluid passes from one section to another, some energy addition or subtraction

Friction

→ Friction (viscous effect). Reduces dimensionless parameter $\frac{V}{\sqrt{g}}$ (i.e. Re).

→ Due to friction, reduction of shear stress, the

$$① \rightarrow \text{Friction} = -\frac{1}{2} \cdot \frac{1}{\rho} \cdot \frac{V^2}{2g} \cdot C_f$$

→ Energy Equation / First Law of Thermodynamics

when we apply energy conservation of the 1st law of thermodynamics to a moving fluid, the statement changes to:

$$\left\{ \begin{array}{l} \text{Rate of change} \\ \text{of energy inside} \\ \text{the fluid element} \end{array} \right\} = \left\{ \begin{array}{l} \text{Net force} \\ \text{of heat} \\ \text{into the} \\ \text{element} \end{array} \right\} + \left\{ \begin{array}{l} \text{Rate of work done} \\ \text{on the element due} \\ \text{to dry g &} \\ \text{surface forces} \end{array} \right\}$$

i.e.

$$\frac{dE}{dt} = \frac{\delta Q}{dt} - \frac{\delta W_{\text{net}}}{dt} \quad (I)$$

∴ δQ & δW are path fⁿ, hence δ is used. (Inexact differential)

E is a point fⁿ, hence d is used

from Reynolds's Transportation Theorem, we know that:

$$\frac{dN}{dt}|_{CM} = \frac{\partial}{\partial t} \iiint_{CV} \rho n \, dV + \iint_{CS} \rho \cdot n \vec{V} \cdot d\vec{A} \quad (I)$$

$$N = \text{extensive property} = E = m \cdot e. \quad (2)$$

where $e = \text{specific energy} = \frac{1}{2} v^2 + \rho g z + \frac{1}{2} \rho w^2 = \frac{1}{2} \rho v^2 + \rho g z$

\downarrow
 ~~ρ~~ = energy per unit mass
 ~~ρ~~ = $(1 + \frac{v^2}{2} + gz)$ (Bernoulli's eq (1))

where ρ = internal energy per unit mass

$$v^2/2 = \text{kinetic energy per unit mass}$$

$$gz = \text{Potential energy per unit mass}$$

$$\gamma = \frac{E}{m} = e. = \text{extensive property per unit mass} \quad (3)$$

∴ from (1) (2) (3)

$$\frac{dE}{dt}|_{CM} = \frac{\partial}{\partial t} \iiint_{CV} \rho e \, dV + \iint_{CS} \rho e \vec{V} \cdot d\vec{A} \quad (II)$$

\Rightarrow ~~total energy~~ \Rightarrow ~~internal + kinetic + potential~~ \Rightarrow ~~total~~ \Rightarrow ~~total~~

also.

$$\frac{dq}{dt} = \frac{dq}{dm} \cdot \frac{dm}{dt} = m \cdot \frac{dq}{dm} = mq. \quad \text{III}$$

also.

$$W_{\text{net}} = W_{\text{sh}} + W_{\text{press}} + W_{\text{reac}} + W_{\text{other}} = \text{net work done on element}$$

$$W_s = \text{Wshaft} - \textcircled{i}$$

$$W_{\text{pres}} = W_{\text{pressure diff.}} = \iint_P P dA ds \quad \text{--- (ii)}$$

$$W_{\text{shear}} = \left(\vec{V} \times d\vec{A} \right) \cdot \vec{V} = W_{\text{shear force}}.$$

where $T = \text{clen}(\sigma)$ (meaning has more than 1 direction
hence 2 arrows on its head)

- also $\vec{C} \times d\vec{A}$ = vector as a tensor \times vector = vector.

\therefore Where $\omega = 0$ if a) $\vec{v} = 0$ for incompressible flow? — (iii)
 , b) $d\vec{A} \perp^r \vec{v}$ —

$$S_{W\text{net}} = S_{Ws} + S_{W\text{pres}} + S_{W\text{leakage}}$$

$$= S w_s + \iint_{CS} P dA - d\vec{S} + 0 \quad \text{Since net electric field is zero}$$

$$\therefore \frac{S_{W\text{net}}}{dt} = \frac{S_{WS}}{dt} + \iint_{CS} P d\vec{A} \cdot \frac{d\vec{s}}{dt} / q = \frac{S_{WS}}{dt} + \iint_{CS} P d\vec{A} \cdot \vec{V}$$

$$= \frac{S_{WS}}{dm} \cdot \frac{dm}{dt} + \iint_{CS} P d\vec{A} \cdot \frac{d\vec{s}}{dt}$$

$$\frac{\delta w_{net}}{dt} = \text{in w.s.} + \iint_{C.S.} P d\vec{A} \cdot \vec{v} \quad \text{--- IV}$$

from (I) (II) (III) (IV)

$$\frac{\partial}{\partial t} \iiint_{CV} \rho e dV + \iint_{CS} \rho e \vec{v} \cdot d\vec{A} = m_q - m_w - \iint_{CS} \rho d\vec{A} \cdot \vec{v}$$

for steady flow $\frac{d}{dt} \iiint_{C-V} \rho dV = 0$

$$\therefore \dot{m}q = \dot{m}w_s + \iint_{CS} P e \vec{V} \cdot d\vec{A} + \iint_{CS} P d\vec{A} \cdot \vec{V}$$

assuming constant properties and \vec{V}

$$\therefore V = 1/\rho = \text{specific volume}$$

$$\& PV = 1 \& \therefore \vec{A} \cdot \vec{B} = \vec{B} \cdot \vec{A}$$

$$\therefore \dot{m}q = \dot{m}w_s + \iint_{CS} e e \vec{V} \cdot d\vec{A} + \iint_{CS} P e V (\vec{V} \cdot d\vec{A})$$

$$\therefore q = w_s + \frac{1}{m} \left[\iint_{CS} e (e \vec{V} \cdot d\vec{A}) + \iint_{CS} P e V (\vec{V} \cdot d\vec{A}) \right]$$

$$q = w_s + \frac{1}{m} \iint_{CS} e dm + \frac{1}{m} \iint_{CS} P e dm$$

$$q = w_s + \frac{1}{m} \iint_{A_1 \rightarrow A_2} (e + P \nu) dm \quad \left. \begin{array}{l} A_2 \rightarrow \text{state 2} \\ A_1 \rightarrow \text{state 1} \end{array} \right\} \text{assuming properties at a section are uniform.}$$

$$\therefore q = w_s + \left[e + \frac{P}{\rho} \right]_{A_1}^{A_2}$$

$$\text{Enthalpy} = \underline{S_e + \nu \vec{V} \cdot \vec{g}}$$

$$q = w_s + \left(e_2 + \frac{P_2}{\rho_2} \right) - \left(e_1 + \frac{P_1}{\rho_1} \right)$$

$$\therefore q + \left(e_1 + \frac{P_1}{\rho_1} \right) = w_s + \left(e_2 + \frac{P_2}{\rho_2} \right)$$

$$q + \left[u_1 + \frac{v_1^2}{2} + gz_1 + \frac{P_1}{\rho_1} \right] = w_s + \left[u_2 + \frac{v_2^2}{2} + gz_2 + \frac{P_2}{\rho_2} \right] \quad \text{--- (1)}$$

$$\therefore \text{enthalpy} = h = u + PV \Rightarrow \& V = 1/\rho$$

$$\left. \begin{array}{l} h_1 = u_1 + \frac{P_1}{\rho_1} \\ h_2 = u_2 + \frac{P_2}{\rho_2} \end{array} \right\} \rightarrow \text{putting in (1)}$$

$$\boxed{h_1 + \frac{v_1^2}{2} + gz_1 + q = w_s + h_2 + \frac{v_2^2}{2} + gz_2}$$

→ deriving Bernoulli's eq. using energy conservation

(Q1) 1st law of thermodynamics :-

$$h_1 + \frac{v_1^2}{2} + gz_1 + q = w_s + h_2 + \frac{v_2^2}{2} + gz_2$$

$$\therefore U_1 + \frac{P_1}{\rho_1} + \frac{v_1^2}{2} + gz_1 + q = w_s + U_2 + \frac{P_2}{\rho_2} + \frac{v_2^2}{2} + gz_2$$

$$\therefore \frac{P_1}{\rho_1} + \frac{v_1^2}{2} + gz_1 = \frac{P_2}{\rho_2} + \frac{v_2^2}{2} + gz_2 + [U_2 - q + w_s - U_1]$$

$$\frac{P_1}{\rho_1} + \frac{v_1^2}{2} + gz_1 = \frac{P_2}{\rho_2} + \frac{v_2^2}{2} + gz_2 + [w_s - q + w_s]$$

$$\therefore \frac{P_1}{\rho_1} + \frac{v_1^2}{2} + gz_1 = \frac{P_2}{\rho_2} + \frac{v_2^2}{2} + gz_2$$

$$\boxed{\frac{P}{\rho} + \frac{v^2}{2} + gz = \text{constant}}$$