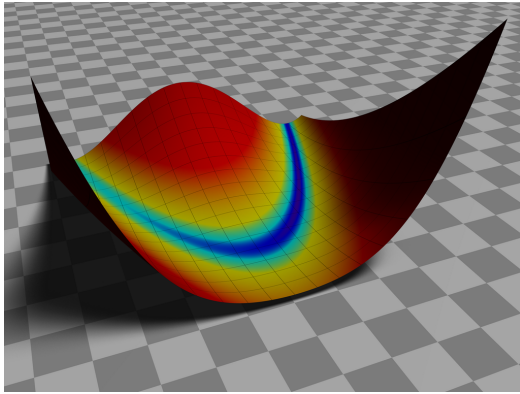


# Direct/Adjoint Methods

## Lecture 11



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## Outline

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Analytic Sensitivity Equations

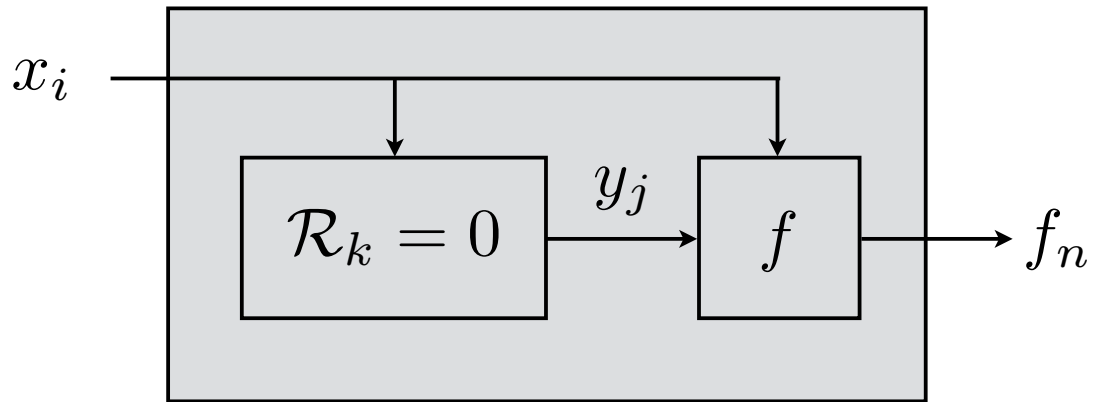
Direct/Adjoint

$x_i$  : design variables

$y_j$  : state variables

$\mathcal{R}_k$  : residuals

$f_n$  : outputs (objectives and constraints)



Our end goal is to get

$$\frac{df_n}{dx_i}$$

for all  $i$  and  $n$ .

$$f_n = f(x_i, y_j(x_i))$$

$$\frac{df_n}{dx_i} = \frac{\partial f_n}{\partial x_i} + \frac{\partial f_n}{\partial y_j} \frac{dy_j}{dx_i}$$

$$\mathcal{R}(x_i, y_j(x_i)) = 0$$

$$\frac{d\mathcal{R}_k}{dx_i} = \frac{\partial \mathcal{R}_k}{\partial x_i} + \frac{\partial \mathcal{R}_k}{\partial y_j} \frac{dy_j}{dx_i} = 0$$

Two equations:

$$\frac{df_n}{dx_i} = \frac{\partial f_n}{\partial x_i} + \frac{\partial f_n}{\partial y_j} \boxed{\frac{dy_j}{dx_i}}$$

$$\frac{d\mathcal{R}_k}{dx_i} = \frac{\partial \mathcal{R}_k}{\partial x_i} + \frac{\partial \mathcal{R}_k}{\partial y_j} \boxed{\frac{dy_j}{dx_i}} = 0$$

Rearrange second equation:

$$\frac{\partial \mathcal{R}_k}{\partial y_j} \frac{dy_j}{dx_i} = - \frac{\partial \mathcal{R}_k}{\partial x_i}$$

Sub into first:

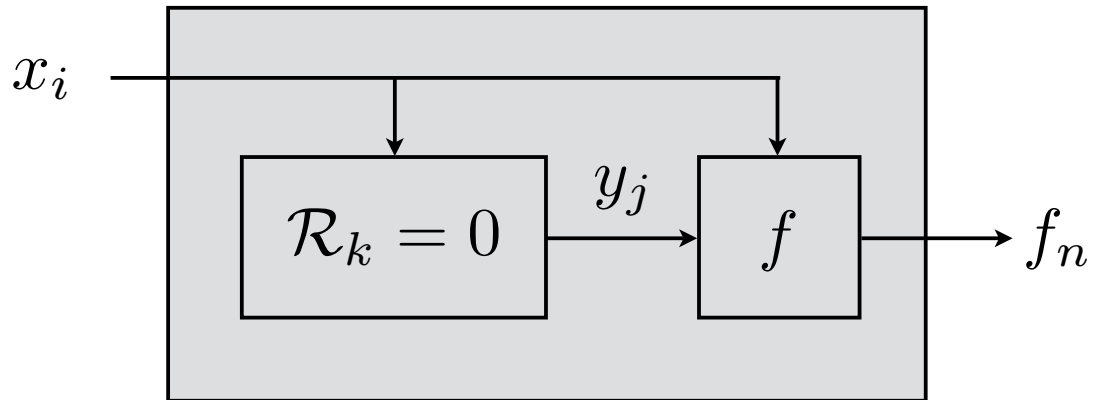
$$\frac{df_n}{dx_i} = \frac{\partial f_n}{\partial x_i} - \frac{\partial f_n}{\partial y_j} \left[ \frac{\partial \mathcal{R}_k}{\partial y_j} \right]^{-1} \frac{\partial \mathcal{R}_k}{\partial x_i}$$

# Analytic Sensitivity Equations

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$$\frac{df_n}{dx_i} = \frac{\partial f_n}{\partial x_i} - \frac{\partial f_n}{\partial y_j} \left[ \frac{\partial \mathcal{R}_k}{\partial y_j} \right]^{-1} \frac{\partial \mathcal{R}_k}{\partial x_i}$$

- Can get total derivatives from partial derivatives that are much more easily obtained.
- We don't actually invert a matrix. We often don't even store the factorization.
- Order of operations is extremely important (more on this next).



Two ways to solve. Partial derivatives are always the same, but **order of operations** is not.

$$\frac{df_n}{dx_i} = \frac{\partial f_n}{\partial x_i} - \underbrace{\frac{\partial f_n}{\partial y_j}}_{\Psi} \underbrace{\left[ \frac{\partial \mathcal{R}_k}{\partial y_j} \right]^{-1}}_{\Phi} \frac{\partial \mathcal{R}_k}{\partial x_i}$$

Direct method:

$$\frac{\partial \mathcal{R}_k}{\partial y_j} \Phi_j = -\frac{\partial \mathcal{R}_k}{\partial x_i}$$

then:

$$\frac{df_n}{dx_i} = \frac{\partial f_n}{\partial x_i} + \frac{\partial f_n}{\partial y_j} \Phi_j$$

**One** linear solve for each input  $x_i$ ,  
but can reuse for **all** outputs  $f_n$

Adjoint method:

$$\left[ \frac{\partial \mathcal{R}_k}{\partial y_j} \right]^T \Psi_k = -\frac{\partial f_n}{\partial y_j}.$$

then

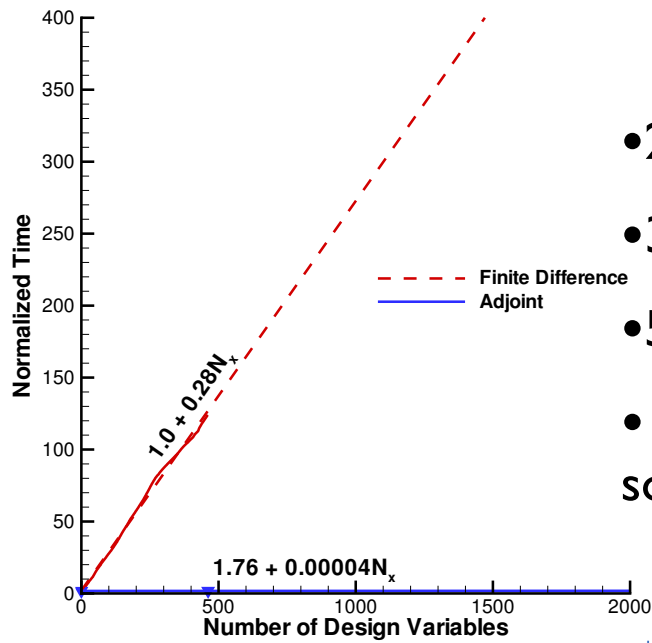
$$\frac{df_n}{dx_i} = \frac{\partial f_n}{\partial x_i} + \Psi_k^T \frac{d\mathcal{R}_k}{dx_i}$$

**One** linear solve for each output  $f_n$ ,  
but can reuse for **all** inputs  $x_i$

Step	Direct	Adjoint
Matrix Factorization	same	same
Back-solve	$N_x$ times	$N_f$ times
Multiplication	same	same

**Important implication:** For the adjoint method, computing gradients is practically independent of the number of design variables.





- 2M CFD cells
- 300k CSM DOFs
- 56 processors
- 1 aerostructural solution = 5.5 min

Kenway, Kennedy and Martins, AIAA Journal, 2012

## Example: Finite Element Analysis

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Force-displacement relationship:

$$F = Ku$$

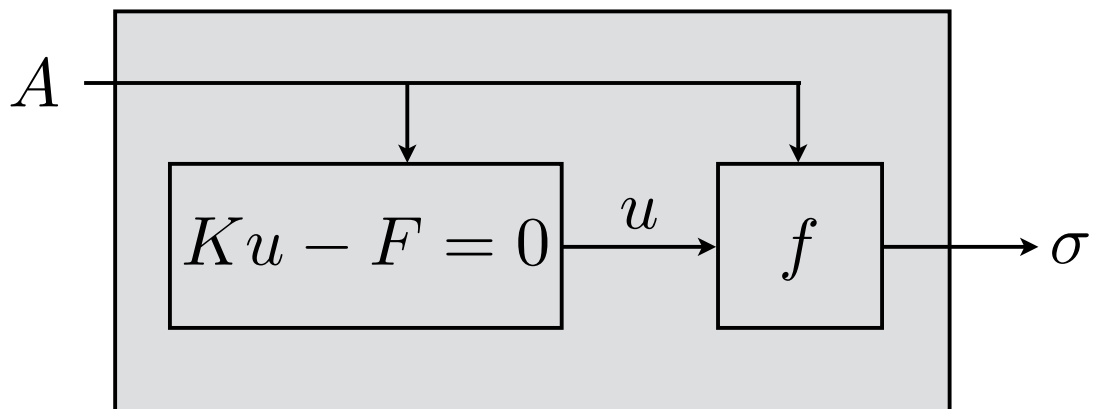
Residual form:

$$\mathcal{R}(u) = Ku - F = 0$$

Stress:

$$\sigma = Su$$

Can change cross-sectional areas of truss.



$x_i$  : design variables

$y_j$  : state variables

$\mathcal{R}_k$  : residuals

$f_n$  : outputs (objectives and constraints)

$A$  : design variables (cross-sectional areas)

$y_j$  : state variables

$\mathcal{R}_k$  : residuals

$f_n$  : outputs (objectives and constraints)

$A$  : design variables (cross-sectional areas)

$u$  : state variables (deflections)

$\mathcal{R}_k$  : residuals

$f_n$  : outputs (objectives and constraints)

$A$  : design variables (cross-sectional areas)

$u$  : state variables (deflections)

$Ku - F = 0$  : residuals (stiffness relationship)

$f_n$  : outputs (objectives and constraints)

$A$  : design variables (cross-sectional areas)

$u$  : state variables (deflections)

$Ku - F = 0$  : residuals (stiffness relationship)

$\sigma$  : outputs (stress)

Partial derivatives:

$$\frac{\partial f}{\partial x} =$$

$$\frac{\partial f}{\partial y} =$$

$$\frac{\partial \mathcal{R}}{\partial x} =$$

$$\frac{\partial \mathcal{R}}{\partial y} =$$

Partial derivatives:

$$\frac{\partial f}{\partial x} = \frac{\partial \sigma}{\partial A} = 0$$

$$\frac{\partial f}{\partial y}$$

$$\frac{\partial \mathcal{R}}{\partial x}$$

$$\frac{\partial \mathcal{R}}{\partial y}$$

Partial derivatives:

$$\frac{\partial f}{\partial x} = \frac{\partial \sigma}{\partial A} = 0$$

$$\frac{\partial f}{\partial y} = \frac{\partial \sigma}{\partial u} = S$$

$$\frac{\partial \mathcal{R}}{\partial x}$$

$$\frac{\partial \mathcal{R}}{\partial y}$$

Partial derivatives:

$$\frac{\partial f}{\partial x} = \frac{\partial \sigma}{\partial A} = 0$$

$$\frac{\partial f}{\partial y} = \frac{\partial \sigma}{\partial u} = S$$

$$\frac{\partial \mathcal{R}}{\partial x} = \frac{\partial \mathcal{R}}{\partial A} = \left[ \frac{\partial K}{\partial A} \right] u$$

$$\frac{\partial \mathcal{R}}{\partial y}$$

Partial derivatives:

$$\frac{\partial f}{\partial x} = \frac{\partial \sigma}{\partial A} = 0$$

$$\frac{\partial f}{\partial y} = \frac{\partial \sigma}{\partial u} = S$$

$$\frac{\partial \mathcal{R}}{\partial x} = \frac{\partial \mathcal{R}}{\partial A} = \left[ \frac{\partial K}{\partial A} \right] u$$

$$\frac{\partial \mathcal{R}}{\partial y} = \frac{\partial \mathcal{R}}{\partial u} = K$$

$$\frac{df_n}{dx_i} = \frac{\partial f_n}{\partial x_i} - \frac{\partial f_n}{\partial y_j} \left[ \frac{\partial \mathcal{R}_k}{\partial y_j} \right]^{-1} \frac{\partial \mathcal{R}_k}{\partial x_i}$$

$$\frac{d\sigma}{dA} = \frac{\partial \sigma}{\partial A} - \frac{\partial \sigma}{\partial u} \left[ \frac{\partial \mathcal{R}_k}{\partial u} \right]^{-1} \frac{\partial \mathcal{R}_k}{\partial A}$$

$$\frac{d\sigma}{dA_i} = -SK^{-1} \left[ \frac{\partial K}{\partial A_i} \right]_u$$