# Self-Driving Cars and Optimal Trajectory

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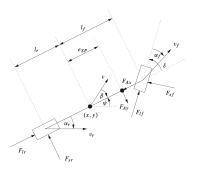
#### Overview

- Single track model
- Optimal control problem formulation
- Road geometry construction
- Alternate optimal control problem formulation using Lagrange collocation
- Model predictive control



# Single Track model

# Single track model



v: velocity

 $I_r$ ,  $I_f$ : distance from COG to rear and front wheels

 $F_{ST}, F_{Sf}$ : lateral tire forces

 $F_{lr}$ ,  $F_{lf}$ : longitudnal tire forces

 $F_{Ax}, F_{Ay}$ : Air drag

 $\alpha_f, \alpha_r, \beta$ : side slip

angles

 $\psi\colon \mathsf{yaw}$  angle

 $\omega_{\mathcal{Z}} :$  angular velocity of COG

 $\delta$ : steering angle

# ODEs for Single Track model

$$\begin{split} \dot{x} &= v cos(\psi - \beta) \\ \dot{y} &= v sin(\psi - \beta) \\ \dot{v} &= \frac{1}{m} \bigg[ (F_{lr} - F_{Ax}) cos\beta - (F_{sr} - F_{Ay}) sin\beta + F_{lf} cos(\delta + \beta) - F_{sf} sin(\delta + \beta) \bigg] \\ \dot{\beta} &= \omega_z - \frac{1}{mv} \bigg[ (F_{lr} - F_{Ax}) sin\beta + (F_{sr} - F_{Ay}) cos\beta + F_{lf} sin(\delta + \beta) + \\ F_{sf} cos(\delta + \beta) \bigg] \\ \dot{\psi} &= \omega_z \\ \dot{\omega}_z &= \frac{1}{I_{zz}} \bigg[ F_{sf} . I_f . cos\delta - F_{sr} . I_r + F_{sf} . I_f . sin\delta \bigg] \\ \dot{\delta} &= \omega_\delta \end{split}$$

# Description of single track model forces

The longitudual tire forces are related to the braking/accelerating forces and the rolling resistances at the tires by

$$F_{lf} = -F_{Bf} - F_{Rf}$$
$$F_{lr} = -F_{Br} - F_{Rr}$$

We assume that the breaking/accelerating force  $F_B$  is distributed as

$$F_{Bf} = egin{cases} rac{\epsilon}{3}F_B, & F_B > \Delta \ rac{5}{6}F_B - rac{1}{4\Delta}F_B^2 + rac{1}{12\Delta^3}F_B^4, & |F_B| \leq \Delta \ F_B, & ext{otherwise} \end{cases}$$
  $F_{Br} = egin{cases} rac{1}{3}, F_B & F_B > \Delta \ rac{2}{3\Delta}F_B^2 - rac{1}{3\Delta^2}F_B^3, & 0 < F_B \leq \Delta \ 0, & ext{otherwise} \end{cases}$ 

where  $\Delta = 0.01$ .  $F_B > 0$  corresponds to acceleration and  $F_B < 0$ corresponds to braking.

# Description of single track model forces

Side slip angles at the front and rear wheels are given by,

$$lpha_{\it f} = \delta - arctanigg(rac{l_{\it f}.\dot{\psi} - v sineta}{v coseta}igg)$$
 $lpha_{\it f} = arctanigg(rac{l_{\it f}.\dot{\psi} + v sineta}{v coseta}igg)$ 

The lateral forces on the tires are given by Pacejka's magic formula [11]

$$F_{sf}(\alpha_f) = D_f.sin(C_f arctan(B_f \alpha_f - E_f(B_f \alpha_f - arctan(B_f \alpha_f))))$$
  
 $F_{sr}(\alpha_r) = D_r.sin(C_r arctan(B_r \alpha_r - E_r(B_r \alpha_r - arctan(B_r \alpha_r))))$ 

# Description of single track model forces

The air drag along the lateral direction of the vehicle is assumed to be zero and along the longitudnal direction it is given by

$$F_{Ax} = rac{1}{2}c_{\omega}
ho Av^2 \ F_{Ay} = 0$$



# Optimal Control problem formulation

# Optimal Control problem formulation

# What is optimal control?

To optimize the controls and state trajectories for a dynamical system over a period of time.

#### How does it work?

Optimize the cost function with respect to control variables while satisfying the ODEs, control & state constraints.



# Defining variables

#### State variables

- x: displacement in x direction
- y: displacement in y direction
- v: velocity of car
- $\triangleright$   $\delta$ : steering angle
- $\blacktriangleright \psi$ : yaw angle
- $\triangleright \ \omega_z$ : angular velocity of COG

#### **Control Variables**

- $\triangleright \omega_{\delta}$ : steering velocity
- $\triangleright$   $F_B$ : breaking or acceleration force

# Defining Variables

State variables -

$$z: t \to \mathbb{R}^7, z(t) = (x, y, v, \beta, \psi, \omega_z, \delta)^T(t)$$

Control Variables -

$$u: t \to \mathbb{R}^2, u(t) = (w_\delta, F_B)^T(t)$$





#### **ODE** constraints

# dynamics of states z is described by, $f: \mathbb{R} \times \mathbb{R}^7 \times \mathbb{R}^2 \to \mathbb{R}^7$

$$\begin{split} \dot{z}(t) &= f(t,z(t),u(t)) \\ &= \begin{bmatrix} v\cos(\psi-\beta) \\ v\sin(\psi-\beta) \\ \frac{1}{m} \Big[ (F_{lr}-F_{Ax})\cos\beta - (F_{sr}-F_{Ay})\sin\beta + F_{lf}\cos(\delta+\beta) - F_{sf}\sin(\delta+\beta) \Big] \\ \omega_{Z} &= \frac{1}{mv} \Big[ (F_{lr}-F_{Ax})\sin\beta + (F_{sr}-F_{Ay})\cos\beta + F_{lf}\sin(\delta+\beta) + F_{sf}\cos(\delta+\beta) \Big] \\ \frac{\omega_{Z}}{l_{ZZ}} \Big[ F_{sf}.l_{f}.\cos\delta - F_{sr}.l_{r} + F_{sf}.l_{f}.\sin\delta \Big] \\ \omega_{\delta} \end{split}$$

# Optimal Control Problem formulation

Objective function: final time

tf

Control constraints -:

$$c(t, z(t), u(t)) := \begin{bmatrix} w_{\delta}(t) - 0.5 \\ -0.5 - w_{\delta}(t) \\ -5000 - F_{B}(t) \\ F_{B}(t) - 15000 \end{bmatrix} \le 0_{\mathbb{R}^{4}}$$
 (1)

state constraints.

$$s(t,z(t)) := \begin{bmatrix} y(t) - y_{up}(x(t)) \\ y_{down}(x(t)) - y(t) \end{bmatrix} \le 0_{\mathbb{R}^2}$$
 (2)

where  $y_{up}$ ,  $y_{down}$  are functions describing road boundary y-coordinates

#### Final and Initial conditions constraints

$$\gamma( extit{z}(t_0), extit{z}(t_f)) := egin{bmatrix} x(t_0) \ y(t_0) \ v(t_0) \ \omega_{ extit{z}}(t_0) \ \delta(t_0) \ x(t_f) - x_{ extit{END}} \end{bmatrix} = 0^{\mathbb{R}^7}$$

- ▶ t<sub>0</sub>: Initial time
- $ightharpoonup t_f$ : Final time

# Converting the OCP to a fixed time problem

Assuming  $t_0 = 0$ , the final time  $t_f$  is free so we use the linear time transformation. To convert the OCP to a fixed time problem we use the following transformation [7]

$$t(\tau) = \tau t_f, \tau \in [0, 1] \tag{3}$$

and introduce the state  $t_f$  which is constant for  $\tau \in [0,1]$  and

$$\frac{d}{d\tau}t_f(\tau)=0$$

The states z are transformed by

$$\bar{\mathit{Z}}( au) := \mathit{Z}( au \mathit{t}_{\mathit{f}})$$

$$\bar{u}(\tau) :== u(\tau t_f)$$

# Converting the OCP to a fixed time problem

Then,

$$\frac{d}{d\tau}\bar{z}(\tau) = \dot{z}(t(\tau)).\frac{d}{d\tau}(t(\tau))$$

$$= f(t(\tau), z(t(\tau))., u(t(\tau))).t_f$$

$$= t_f f(\tau t_f, \bar{z}(\tau), \bar{u}(\tau))$$

Now we redefine  $\bar{z}(\tau) = (\bar{z}(\tau), t_f(\tau))$  to include the new state  $t_f$  Then the optimization problem is given by,

# Optimal Control Problem formulation

#### Minimize

 $t_f$ 

with respect to  $\bar{z}$  and  $\bar{u}$ , subject to the ODEs,

$$\dot{\bar{z}}(t) = f\bigg(\tau t_f, \bar{z}(t), \bar{u}(t)\bigg)\bigg)$$

and constraints,

$$ar{c}igg( au t_f,ar{z}(t),ar{u}(t)igg):=c(t( au),z(t( au)),u(t( au)))\leq 0_{\mathbb{R}^6} \ ar{s}igg( au t_f,ar{z}(t),ar{u}(t)igg):=s(t( au),z(t( au)),u(t( au)))\leq 0_{\mathbb{R}^2} \ ar{\gamma}(ar{z}_0,ar{z}_N)=0_{\mathbb{R}^7}$$

#### Full Discretization for states and controls

To include the ODE constraints we use the full discretization method. [7]

We define a grid with N steps,

$$G_N = \{ \tau_i | i = 0, 1, 2, 3, ..., N \}$$

We use an ODE method **ode** which given the states at  $\bar{z}_j$  computes the states at  $\bar{z}_{j+1}$  such that

$$\bar{z}_{j+1} = ode(\bar{z}_j, h), j = 0, 1, \dots, N-1 \text{ and } h = \frac{1}{N}$$
 (4)

We discretize the control space with piecewise linear functions on the grid  $G_N$  to get  $\{m_0, m_1, \dots, m_{N-1}\}$  controls where  $m_i \in \mathbb{R}^2$ 



# OCP formulation using full discretization

Then the discretized optimisation problem is, Minimize

$$t_f(N)$$

with respect to  $(\bar{z}_0, \bar{z}_2, \dots, \bar{z}_N, m_0, \dots, m_{N-1}) \in \mathbb{R}^{8N+2(N+1)}$ , subject to,

$$\bar{z}_{j+1} - ode(\bar{z}_j, h) = 0_{\mathbb{R}^8}, \ j = 0, 1, \dots, N-1$$

# OCP formulation using full discretization

and constraints,

$$ar{c}( au_{j}, ar{z}_{j}, ar{u}_{M}( au_{j}; m)) \leq 0_{\mathbb{R}^{6}}, \ j = 0, 1, \dots, N-1 \ ar{s}( au_{j}, ar{z}_{j}) \leq 0_{\mathbb{R}^{2}} \ j = 0, 1, \dots, N-1 \ ar{\gamma}(ar{z}_{0}, ar{z}_{N}) = 0_{\mathbb{R}^{7}}$$

Collecting all the variables and constraints into matrices, we get the optimization problem, Minimise

$$J(\bar{q})=t_f(\tau_N)$$

with respect to  $(\bar{z}_0, \bar{z}_2, \dots, \bar{z}_N, m_1, \dots, m_{N-1}) \in \mathbb{R}^{8(N+1)+2N)}$ , subject to,





#### with constraints

$$G(\bar{q}) = \begin{bmatrix} \bar{c}(\tau_{0}, \bar{z}_{0}, \bar{u}_{M}(\tau_{0}; m)) \\ \bar{c}(\tau_{1}, \bar{z}_{1}, \bar{u}_{M}(\tau_{1}; m)) \\ \vdots \\ \bar{c}(\tau_{N}, \bar{z}_{N}, \bar{u}_{M}(\tau_{N}; m)) \\ \bar{s}(\tau_{0}, \bar{z}_{0}) \\ \bar{s}(\tau_{1}, \bar{z}_{N}) \\ \vdots \\ \bar{s}(\tau_{N}, \bar{z}_{N}) \end{bmatrix} \leq 0_{\mathbb{R}^{8(N+1)+2N}}$$
(5)

and.

$$H(ar{q}) = egin{bmatrix} ar{z}_1 - ode(ar{z}_0,h) \ ar{z}_2 - ode(ar{z}_1,h) \ dots \ ar{z}_N - ode(ar{z}_{N-1},h) \ ar{\gamma}(ar{z}_0,ar{z}_N) \end{bmatrix} = 0_{\mathbb{R}^{8(N+1)+9}} \quad (1)$$

(6)



# Road Geometry Construction

# Road Geometry Construction



### Degrees to Distance

The number of meters one has to travel to move 1 degree in latitude is,

111132.92
$$-559.82\cos(2\phi_{lat})+1.175\cos(4\phi_{lat})-0.0023\cos(6\phi_{lat})$$

The same to move 1 degree in longitude is,

$$111412.84\cos(4\phi_{long}) - 93.5\cos(3\phi_{long}) + 0.118\cos(5\phi_{long})$$

[4]

#### **Cubic Splines**

The points need to represented as,

$$\mathbf{P}_i = (x_i, y_i) = (X_i(u), Y_i(u))$$

where i = 1, ..., N and N is the number of data points. For a cubic polynomial,

$$X_{i}(u) = a_{x,i} + b_{x,i}(u - u_{i+1}) + c_{x,i}(u - u_{i+1})^{2} + d_{x,i}(u - u_{i+1})^{3}$$
  

$$Y_{i}(u) = a_{y,i} + b_{y,i}(u - u_{i+1}) + c_{y,i}(u - u_{i+1})^{2} + d_{y,i}(u - u_{i+1})^{3}$$

where  $u \in [u_i, u_{i+1}]$ .

#### Constraints

We need additional constraints to compute all the coefficients

$$X_{i}(u)^{(1)}\Big|_{u=u_{i+1}} = D_{x,i} = b_{x,i}$$

$$Y_{i}(u)^{(1)}\Big|_{u=u_{i+1}} = D_{y,i} = b_{y,i}$$

$$X_{i}(u)^{(1)}\Big|_{u=u_{i}} = D_{x,i+1} = b_{x,i} + 2c_{x,i}(u_{x,i} - u_{x,i+1}) + 3d_{x,i}(u_{x,i} - u_{x,i+1})$$

$$Y_{i}(u)^{(1)}\Big|_{u=u_{i}} = D_{y,i+1} = b_{y,i} + 2c_{y,i}(u_{y,i} - u_{y,i+1}) + 3d_{y,i}(u_{y,i} - u_{y,i+1})$$

#### Practical computation of constraints

- How to choose D<sub>i</sub>'s?
- Natural splines (special case of Hermite Interpolation) impose

$$X_{i}^{(2)}(u)\bigg|_{u=u_{x,i}} = X_{i+1}^{(2)}(u)\bigg|_{u=u_{x,i+1}}$$
$$Y_{i}^{(2)}(u)\bigg|_{u=u_{y,i}} = X_{i+1}^{(2)}(u)\bigg|_{u=u_{y,i+1}}$$

We have 2(N-2) equations. We need 4 more conditions to compute  $D_i$ 's  $\forall i = 1, ..., N$ 

# End conditions for practical computations

- End conditions
- Clamped or complete, not-a-knot, periodic, . . .
- Choice of the user and based on application.
- MATLAB's csapi from Curve Fitting Toolbox uses not-a-knot end conditions.

$$X_{2}^{(3)}(u)\Big|_{u=u_{x,2}} = X_{N-1}^{(3)}(u)\Big|_{u=u_{x,N-1}} = 0$$

$$Y_{2}^{(3)}(u)\Big|_{u=u_{y,2}} = Y_{N-1}^{(3)}(u)\Big|_{u=u_{y,N-1}} = 0$$

▶ 2N equations to solve for  $D_{x,i}$  and  $D_{y,i}$  for i = 1, ..., N.

(R.H Bartels et al.[3])

#### **Parametrizations**

- How to choose the parameter values (knots), u?
- The parameters are recursively computed by,

$$u_0 = 0, \ u_{i+1} = u_i + d_i$$

where we define

$$d_i := \| \boldsymbol{P}_{i+1} - \boldsymbol{P}_i \|_2^{\mu}$$

 $\mu$  is called the blending parameter. The value of  $\mu$  influences the overall shape [6].

 $\mu = 0 \implies$  uniform parametrization and  $\mu = 1 \implies$  chordal parametrization.

# Normal projection

Coordinates of the boundaries (inner and outer) of the road with width w can be computed by moving in normal direction,

$$\mathbf{P}_{i-inner} = \mathbf{P}_i - \frac{w}{2}\mathbf{\hat{n}}$$

$$\mathbf{P}_{i-outer} = \mathbf{P}_i + \frac{w}{2}\mathbf{\hat{n}}$$

# Road modeling

# Algorithm

- 1. Project GPS coordinates of the road centerline.
- 2. Interpolate the centerline using csapi.
- 3. Sample *M* points from the centerline spline. Compute the coordinates of inner and outer boundaries of the road.
- 4. Interpolate the boundaries of the road.

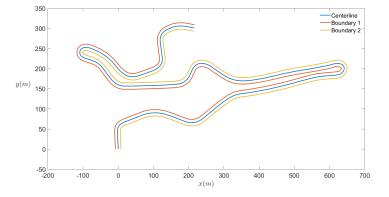


# Insights

# Findings

- 1.  $\mu=0 \implies$  poor modeling of the road boundaries when M>>N. Inner road boundaries have normal crossings and require separate cleaning and/or smoothing procedure.
- 2.  $\mu = 0$  better in the above aspect.
- 3. But for a given M that is close to N,  $\mu = 0 \implies$  relatively better construction of the overall geometry of the road.

#### **Constructed Road**



#### Modified Road Constraints for the OCP

Road constraints

$$s(t, z(t)) := \begin{bmatrix} y(t) - y_{up}(x(t)) \\ y_{down}(x(t)) - y(t) \end{bmatrix} \le 0_{\mathbb{R}^2}$$

Need to modify our state variable ODEs to include the spline parametrization.

$$\dot{x} = \frac{dx}{du}\dot{u} = t_f v_x \tag{7}$$

$$\dot{y} = \frac{dy}{du}\dot{u} = t_f v_y \tag{8}$$

We can replace the dynamics ODEs (Eqs. 7-8) with the dynamics of parameter t

$$\dot{u} = t_f v_X \left(\frac{dx}{du}\right)^{-1} \tag{9}$$

#### Modified Road Constraints for the OCP

#### Modified Road constraints

$$s(t, z(t)) := \begin{bmatrix} y(u) - y_{outer}(u) \\ y_{inner}(u) - y(u) \end{bmatrix} \le 0_{\mathbb{R}^2}$$
 (10)



# Tests on Straight road

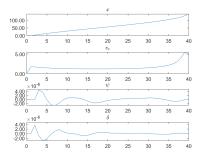


## Tests on Straight road

- Feasibility studies were performed on a straight road.
- ODEs were integrated using Explicit Euler method,
   MATLAB's ode45 [13] and dop54 [1]
- Stiff Solvers like MATLAB's ode15s and radau were also tested - Computationally expensive and were unsuccessful in finding even a feasible point.

# **Explicit Euler**

- ► Total Time steps N = 40
- $x_0 = 0 \ x_{END} = 140 \ \text{m}$
- interior-point in MATLAB's fmincon



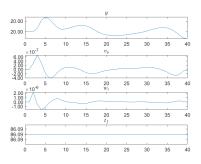


Figure: Feasible State Variables for straight road using Explicit Euler method





## **Explicit Euler**



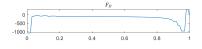
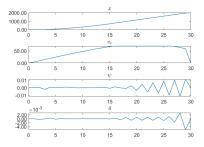


Figure: Feasible Control Variables for straight road using Explicit Euler method

- Feasible solution used as initial guess for optimization.
- Optimal solution not found.



- ▶ Total Time steps N = 30
- $x_0 = 0 \ x_{END} = 2000 \ \text{m}$
- interior-point in MATLAB's fmincon



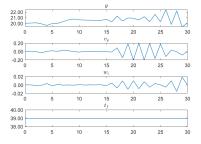
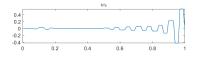


Figure: Feasible State Variables for straight road using ode45







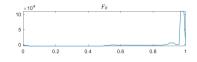
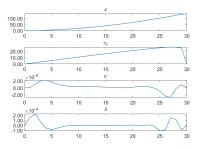


Figure: Feasible Control Variables for straight road using ode45

- ► Feasible solution used as initial guess for optimization.
- Optimal solution not found.

- ► Total Time steps N = 30
- $x_0 = 0 \ x_{END} = 140 \ \text{m}$
- interior-point in MATLAB's fmincon



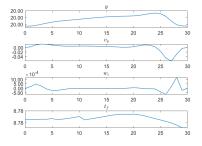


Figure: Feasible State Variables for straight road using dop54





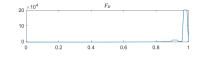


Figure: Feasible Control Variables for straight road using dop54

- Feasible solution used as initial guess for optimization.
- Optimal solution not found.





# Alternate optimal control problem formulation using Lagrange collocation

# Optimal trajectory using Lagrange collocation and new objective formulation

#### Problems with previous formulation

- Relies on explicit ODE solver
- directly minimising time which is included as a state
- optimization steps are slow because of slow ODE solver computation
- some ODE solvers run into discontinuities during optimization

Uses a path coordinate *s* for formulating ODEs and optimization problem. [12]

s: independent variable

 $t_f$ : dependent variable

- s: progress of car along track
- track boundaries, initial and final end points can be specified easily with s
- s not required as state in optimization

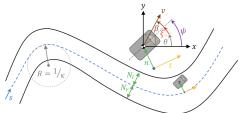
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# New objective formulation

# Introduce new variables [5]

- $\triangleright$  s: parameter for progress along track centerline
- θ : angle between tangent to track centerline and x-direction
- $\triangleright$  n: lateral displacement from track centerline
- $\blacktriangleright$   $\xi$ : angle between vehicle axis and tangent to centerline
- $\triangleright$   $\kappa$ : curvature of track centerline at any point s



Using  $s, n, \xi$  position and orientation of vehicle on the track can be specified

ODEs for the new states  $s, n, \xi$  w.r.t time (t)

$$\dot{s} = \frac{v\cos(\xi - \beta)}{1 - n\kappa(s)} \tag{11}$$

$$\dot{n} = v \sin(\xi - \beta) \tag{12}$$

$$\dot{\xi} = \omega_z - \kappa \frac{v\cos(\xi - \beta)}{1 - n\kappa(s)} \tag{13}$$

#### ODEs for the new formulation

$$\dot{v} = \frac{1}{m} \left[ (F_{lr} - F_{Ax}) \cos\beta - (F_{sr} - F_{Ay}) \sin\beta + F_{lf} \cos(\delta + \beta) - F_{sf} \sin(\delta + \beta) \right]$$

$$\dot{\beta} = \omega_z - \frac{1}{mv} \left[ (F_{lr} - F_{Ax}) \sin\beta + (F_{sr} - F_{Ay}) \cos\beta + F_{lf} \sin(\delta + \beta) + F_{sf} \cos(\delta + \beta) \right]$$

$$\dot{\omega}_z = \frac{1}{I_{zz}} \left[ F_{sf} . I_f . \cos\delta - F_{sr} . I_r + F_{sf} . I_f . \sin\delta \right]$$

$$\dot{s} = \frac{v\cos(\xi - \beta)}{1 - n\kappa(s)}$$

$$\dot{r} = v\sin(\xi - \beta)$$

$$\dot{\xi} = \omega_z - \kappa \frac{v\cos(\xi - \beta)}{1 - n\kappa(s)}$$

The ODEs, constraints and objectives are then modified according to new independent variable *s*. from (1),

$$\frac{dt}{ds} = \frac{1 - n\kappa(s)}{v\cos(\xi - \beta)} := J(s)$$
 (14)

and we get the new objective function by,

$$t_f = \int_{t_0}^{t_f} dt = \int_{s_0}^{s_f} J(s).ds$$
 (15)

where  $s_f$  and  $s_0$  are the values of s at the start and end point of the track.

#### **ODE** transformation

Similarly ODEs for all states are transformed for the variable s, for an arbitrary state x and control u,

$$\dot{x} = \frac{dx}{dt} = f(x(t), u(t))$$

is transformed into

$$\dot{x} = \frac{dx}{ds} = \frac{dx}{dt}\frac{dt}{ds} = J(s).f(x(s), u(s))$$
 (16)

ODEs of the STM and orientation are transformed using (16)

#### New optimal control problem formulation

$$\min_{x,u} \int_{s_0}^{s_f} J(s).ds 
s.t. \frac{dx}{ds} - f(x(s), u(s)) = 0 
g(x(s), u(s)) \le 0 
h(x(s), u(s)) = 0 
\gamma(x(s_0), x(s_f)) = 0$$

with states x, controls u, STM equations for v,  $\beta$ ,  $\omega_z$  and orientation equations for n,  $\xi$  in f, equality constraints in h, inequality constraints in g and initial and final point constraints in  $\gamma$ 

# Discretization of states using Lagrange collocation

Controls and states are discretized on a grid  $s_k$ , k = 0, 1, ..., N with uniform interval  $\Delta s$  where  $\Delta s = \frac{s_f - s_0}{N}$ The states on an interval  $[s_k, s_{k+1}]$  are approximated using Lagrange basis polynomials  $p_{k,i}$ 

$$ar{x}_k(s) = \sum_{i=0}^d heta_{k,i}.p_{k,i}(s), \ s \in [s_k,s_{k+1}]$$
 $p_{k,i}(s) = \prod_{j=0,j 
eq i}^d rac{s-s_{k,j}}{s_{k,i}-s_{k,j}}$ 

Here  $s_{k,i}$  are the collocation points for the interval. We can see that  $x_k(s_{k,i}) = \theta_{k,i}, i = 0, 1, ..., d$ 

# Discretization of states using Lagrange collocation

We use Gauss-Legendre collocation with d=3. The controls are discretized using piecewise constant functions. The transformed optimal control problem is

$$\min_{\theta, u} \sum_{k=0}^{N-1} \sum_{i=0}^{d} J_k(\theta_{k,i}) \quad \text{s.t.}$$
 (17)

$$\dot{\bar{x}}_k(s_{k,j}) - \Delta sf(\theta_{k,j}, u_k) = 0, \ k = 0, \dots, N-1, j = 1, \dots, d$$
 (18)

$$\dot{\bar{x}}_k(s_{k+1,j}) - \theta_{k+1,0} = 0, \ k = 0, \dots, N-1$$
 (19)

$$g(\theta_k, 0, u(k)) \le 0, k = 0, \dots, N-1$$
 (20)

$$h(x(s), u(s)) = 0, k = 0, ..., N-1$$
 (21)

$$\gamma(\theta_{0,0},\theta_{N,0}) = 0 \tag{22}$$

with  $\int_{s_0}^{s_f} J(s).ds = \sum_{k=0}^{N-1} \Delta s \sum_{r=0}^{d} B_r J_k(\theta_{k,r})$  where  $B_r$  is the integral of the Lagrange basis polynomials.



## Optimization for optimal trajectory

We have 6 states  $v, \beta, \omega_Z, ..., \epsilon$  and 3 controls  $\delta, F_{drive}, F_{brake}$  with the additional constraint  $F_{drive}.F_{brake} = 0$ 

Road boundary constraints are included using the smooth interpolation described above.

The decision variables of the optimization are

$$\{\theta_{0,0},\ldots,\theta_{0,d},u_0,\theta_{1,0},\ldots,\theta_{1,d},u_1,\ldots,\theta_{N-1,0},u_{N-1}\}\$$
 where d=3,  $\theta_{k,i} \in \mathbb{R}^6$  and  $u_k \in \mathbb{R}^3$ 

Large number of variables for optimization but analytical solutions of ODEs are explicitly included as equality constraints and ODEs don't require integration by solvers.

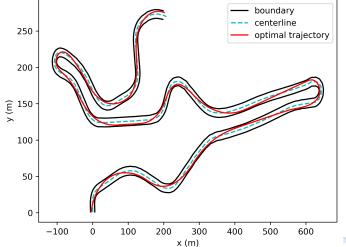
We use Primal-Dual Interior Point method to solve the NLP with the solver IPOPT [14] and discretization interval  $\Delta s = 3m$ .



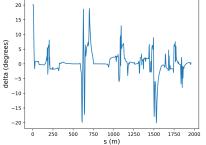
# Optimal trajectory

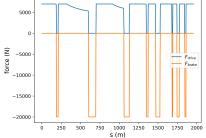
Optimal trajectory time: 78.052 s

Optimization runtime: 19 s

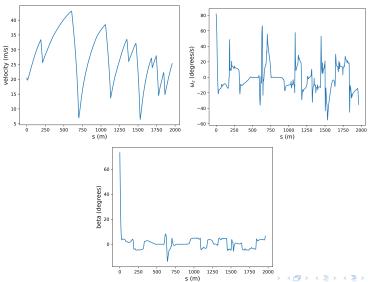


## Optimal controls





# Optimal states



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# Model Predictive Control



#### Model Predictive Control (MPC)

#### What is a MPC?

Optimal control strategy

#### How does MPC work?

► The future response is predicted by predictive control that use a discrete linear time invariant dynamic model

#### Why is MPC good?

 Control linear and nonlinear systems while taking into account state as well as input constraints

#### **MPC** Formulation

$$\begin{aligned} x_{t+1} &= Ax_t + Bu_t \\ y_t &= Cx_t + u_t \\ x_{t+1} &\in \mathbb{R}^n, u_{t+1} \in \mathbb{R}^m, y_t \in \mathbb{R}^p, t \in \mathbb{Z} \end{aligned}$$

with state  $vector x_{t+1}$ , control input  $u_{t+1}$ , output vector  $y_t$  and time index t

#### MPC formulation

$$\min_{u} J(x(t), \mathbf{u})$$
$$\mathbf{u} = (u_t, ... u_{t+N_p-1})$$

#### Subject to

$$x_{t+k+1} = A_{x_t+k} + B_{u_t+k}, \forall k = 0, ....N_p - 1$$
  
 $x_{t+k} \in \mathbb{X}, \forall k = 0, ....N_p - 1$   
 $u_{t+k} \in \mathbb{U}, \forall k = 0, ....N_p - 1$   
 $x_t = x(t)$ 

#### **MPC** Formulation

#### Based on the dynamic ODEs

$$\dot{x} = vcos(\psi - \beta)$$
  
 $\dot{y} = vsin(\psi - \beta)$ 

$$\dot{v} = \frac{1}{m} \left[ (F_{lr} - F_{Ax}) cos\beta - (F_{sr} - F_{Ay}) sin\beta + F_{lf} cos(\delta + \beta) - F_{sf} sin(\delta + \beta) \right]$$

the state space representation is:  $\begin{cases} \dot{x} = f(x(t), u(t)) \\ y_t = g(x(t)) \end{cases}$ with state vector  $\mathbf{x} = [\dot{\mathbf{x}}, \dot{\mathbf{y}}, \dot{\phi}, \phi, X, Y]$ 

# MPC for Self-Driving Car

#### **Motion Control**

Motion Control MPC is based on vehicle kinetic model by finding the next closest point to plan suitable motion.[15]

#### Dynamic Control

Dynamic Control MPC is based on state error composed by heading error, lateral error and location error etc.

station error = 
$$-(dx \cos \theta_{des} + dy \sin \theta_{des})$$
  
speed error =  $V_{des} - V \cos \Delta \theta / k$ 

#### MPC Formulation

With the objective to follow a desired path, the MPC was formulated as[9]:

$$\min \sum_{t=1}^{N_p} Q_1 e_{y_t}^2 + Q_2 e_{\phi_t}^2 + Q_3 e_{\epsilon_t}^2 + Q_4 u_{t-1}^2 + R_1 \Delta u_t^2$$

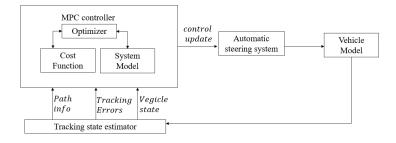
s.t

$$x_{t+1} = Ax_t + B(u_t + \Delta u_t) + C, \forall t = 0, ....N_p$$

$$\begin{aligned} |u_t| - \varepsilon_t &\leq u_{max}, \forall t = 0, .... N_p \\ |\Delta u_{t-1}| &\leq \Delta u_{max}, \forall t = 0, .... N_p \\ \epsilon_t &\geq 0, \forall t = 0, .... N_p \end{aligned}$$

where  $e_{v_t}$  is lateral deviation to the reference path  $e_{\phi_t}$  is the error in the heading angle and  $\epsilon_t$  is a slack variable to soft  $u_t$  from becoming infeasible.

#### MPC Flow Chart





# Basic NMPC algorithm for time varying reference $x^{ref-1}$

At each simpling time  $t_n$ , n = 0, 1, 2, ...

- 1. Measure the state  $x(n) \in X$  of the system
- 2. Set  $x_0 = x(n)$ , solve the optimal control problem

$$\min J_N(n, x_0, u(\cdot)) := \sum_{k=0}^{N-1} \ell(n+k, x_u(k, x_0), u(k))$$
w.r.t  $u(\cdot) \in \mathbb{U}^N(x_0)$ 
subject to  $x_u(0, x_0) = x_0, x_u(k+1, x_0) = f(x_u(k, x_0), u(k))$ 

and denote the obtained optimal control sequence by  $u^*(\cdot) \in \mathbb{U}^N(x_0)$ 

3. Define the NMPC-feedback value  $\mu_n(n, x(n)) := u^*(0) \in U$ and use this control value in the next sampling period.

<sup>1 [8]</sup> Lars Grüne and Jürgen Pannek, Nonlinear Model Predictive Control, 2017 🔻 🦪 🕨 < 📃 🕨



Group 1

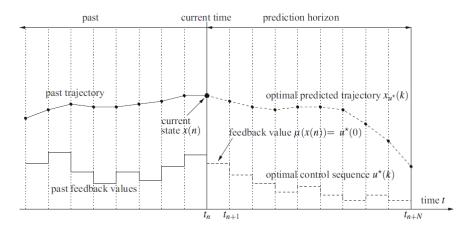


Figure: Illustratin of the NMPC step at time  $t_n$ <sup>2</sup>

<sup>&</sup>lt;sup>2</sup>[8] Lars Grüne and Jürgen Pannek, Nonlinear Model Predictive Control, 2017 ▶ ◀ 🗗 ▶ ◀ 臺 ▶ ◀ 臺 ▶ 🧸 🛫 🔍 🤇

#### From OCP to NLP

$$\begin{bmatrix} \dot{X} \\ \dot{y} \\ \dot{\theta} \end{bmatrix} = \begin{bmatrix} v \cos \theta \\ v \sin \theta \\ \omega \end{bmatrix} = \begin{bmatrix} \cos \theta & 0 \\ \sin \theta & 0 \\ 0 & 1 \end{bmatrix} \cdot \begin{bmatrix} v \\ \omega \end{bmatrix}$$

system model:

$$\dot{\boldsymbol{x}}(t) = \boldsymbol{f}(\boldsymbol{x}_{\boldsymbol{u}}(k), \boldsymbol{u}_{k}) \qquad \boldsymbol{x}(k+1) = \boldsymbol{f}(\boldsymbol{x}(k), \boldsymbol{u}_{k}) \\
\begin{bmatrix} \dot{\boldsymbol{x}} \\ \dot{\boldsymbol{y}} \\ \dot{\boldsymbol{\theta}} \end{bmatrix} = \begin{bmatrix} \boldsymbol{v} \cos \theta \\ \boldsymbol{v} \sin \theta \\ \omega \end{bmatrix} \xrightarrow{\text{Euler Discretization} \\ \text{Sampling Time } \boldsymbol{\Delta T}} \quad \begin{bmatrix} \boldsymbol{x}(k+1) \\ \boldsymbol{y}(k+1) \\ \theta(k+1) \end{bmatrix} = \begin{bmatrix} \boldsymbol{x}(k) \\ \boldsymbol{y}(k) \\ \theta(k) \end{bmatrix} + \boldsymbol{\Delta} T \begin{bmatrix} \boldsymbol{v}(k) \cos \theta(k) \\ \boldsymbol{v}(k) \sin \theta(k) \\ \omega(k) \end{bmatrix}$$

running costs: 
$$\ell(\boldsymbol{x}, \boldsymbol{u}) = \|\boldsymbol{x}_{\boldsymbol{u}} - \boldsymbol{x}^{\text{ref}}\|_Q^2 + \lambda \|\boldsymbol{u} - \boldsymbol{u}^{\text{ref}}\|_R^2$$

cost function: 
$$J_N(\boldsymbol{x}_0, \boldsymbol{u}) = \sum_{k=0}^{N-1} \ell(\boldsymbol{x}_{\boldsymbol{u}}(k), \boldsymbol{u}(k))$$

value function: 
$$V_N(x) = \min_{u} J_n(\boldsymbol{x}_0, \boldsymbol{u})$$

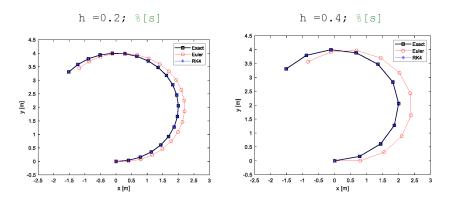


Figure: Comparison between Euler and RK4<sup>3</sup>

<sup>&</sup>lt;sup>3</sup>[10] Mohamed W. Mehrez, Prediction Model Simulation Using Runge Kutta Method, 2020 📑 🛌 📑 🔻

## Runge-Kutta 4th

$$\dot{x} = f(x, u(t))$$

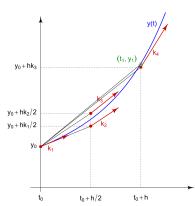
$$k_1 = f(x_{n-1}, u)$$

$$k_2 = f(x_{n-1} + \frac{h}{2}k_1, u)$$

$$k_3 = f(x_{n-1} + \frac{h}{2}k_2, u)$$

$$k_4 = f(x_{n-1} + hk_3, u)$$

$$x_n = x_{n-1} + \frac{h}{6}(K_1 + 2k_2 + 2k_3 + k_4)$$



wikipedia: Runge-Kutta Method



### From OCP to NLP

$$\begin{array}{c} \textbf{OCP} \xrightarrow{\text{Shooting Methods}} \textbf{NLP} \\ \hline \text{Euler, RK4} \end{array}$$

OCP

$$\min_{\boldsymbol{U}} J_{N}(\boldsymbol{x}_{0}, \boldsymbol{u}) = \sum_{k=0}^{N-1} \ell(\boldsymbol{x}_{\boldsymbol{U}}(k), \boldsymbol{u}(k))$$
subject to  $\boldsymbol{x}_{\boldsymbol{U}}(k+1) = \boldsymbol{f}(\boldsymbol{x}_{\boldsymbol{U}}(k), \boldsymbol{u}_{k}),$ 

$$\boldsymbol{X}_{\boldsymbol{U}}(0) = \bar{\boldsymbol{x}}_{0},$$

$$\boldsymbol{u}(k) \in U, \forall k \in [0, N-1]$$

$$\boldsymbol{x}_{\boldsymbol{U}}(k) \in X, \forall k \in [0, N]$$

#### From OCP to NLP

**NLP** with 
$$\mathbf{w} = [\mathbf{u}_0 \cdots \mathbf{u}_{N-1}, \mathbf{x}_0 \cdots \mathbf{x}_N]$$

$$\min_{\mathbf{w}} \Phi(\mathbf{w}) \text{ objective function}$$

subject to 
$$\boldsymbol{g}_1(\boldsymbol{w}) = \begin{bmatrix} g_1(\boldsymbol{x}_0, \boldsymbol{u}_0) \\ \vdots \\ g_1(\boldsymbol{x}_{N-1}, \boldsymbol{u}_{N-1}) \\ g_1(\boldsymbol{x}_N) \end{bmatrix} \le 0, \quad \boldsymbol{g}_2(\boldsymbol{w}) = \begin{bmatrix} \bar{\boldsymbol{x}}_0 - \boldsymbol{x}_0 \\ f(\boldsymbol{x}_0, \boldsymbol{u}_0) - \boldsymbol{x}_1 \\ \vdots \\ f(\boldsymbol{x}_{N-1}, \boldsymbol{u}_{N-1}) - \boldsymbol{x}_N \end{bmatrix} = 0$$

inequality constraints

10.2.2023

equality constraints



- General scope of numerical optimization
- The solution of NI P's
- Facilitates the solution of optimal control problems (OCPs)
  - Write state-of-the-art OCP algorithms with very little code!
- Free & open-source (LGPL), also for commercial use
- Four standard problems can be handled by CasADi
- Supports high-level operations: matrix-operations, implicit functions, calls to DAE integrators
  - Quadratic programs
  - nonlinear programs
  - root finding problems
  - initial-value problems in ODE/DAE

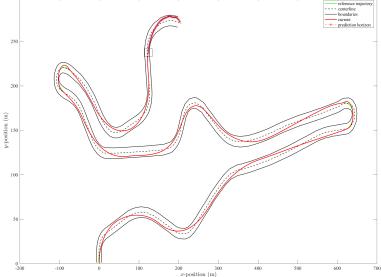
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### Our Result without obstacles

Different choice of prediction horizon (matlab sample).

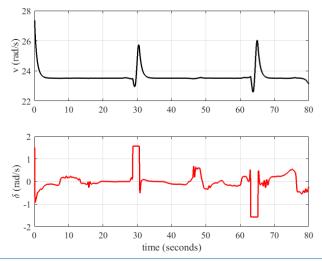
# N=50 in this case.





## **MPC Simulation**

# Control inputs for N=50.



### Future scope

### For Vehicle Dynamics:

- Implement Double Track Model
- Compare the differences

### For optimal trajectory:

- Smoothing for control variables
- Friction maps

#### For MPC:

- Implement more complicated Vehicle Dynamics
- Implement different scenerio
- Proof the stability and robustness of MPC

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