ME 759: Nonlinear FEM Assignment 4: 2-D Elasto-plasticity

Equations used for coding

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06/04/2022
203010005
                Assignment 4-20 Elastoplasticity
Prafull Bhosale
   Variables passed in as information into UMAT by
      Abaque.
                STRAN (NTENS), DSTRACNTENS)
    Solution dependent state variables used in code
       E' = epsilon-plos (NTENS) -- array with
                                        dimension NTENS
       X = alpha & Equivalent plastic strain
   epsilon-plas (1) = STATEV(1)
   epsilon -plas(2) = STATEV(2)
   epsilon-plas (3) = STATEV (3)
   epsilon-plas(4) = STATEV(4)
           alpha = STATEV(5)
- In UMAT, it is necessary to rotate tensors during a
  finite-strain analysis. The matrix DROT that is passed into
  UMAT represents incremental votation of the material
   basis system in which stress & strain one rot stored.
 - For elastic-plastic material that hardens isotropically
    elastic & plastic strain tensors must be rotated to account
    for evolution of material direction
      So, we have to use ROTS I'M inbuilt function incode
        CALL ROTSIG (STATEVUI), DROT, epsilon-plas, 2,
                                      NOT, NSHR
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Algorithm steps:

1) Take material properties from PROPS (...) array.

3) Compute DDSDDE (,) for elastic step.

4) Compute trial stress as

Ontile on (i)+ DDSDDE (i,j) *DSTRAN(j)

5) Compute hydrostatic stress: Thydro- mit $\sigma_H = tr(\sigma_{n+1}) = \sigma(1) + 6(2) + 6(3)$ nti

6) Compute deviatoric stress as 第分3 t = 20t3 - OH {1}

7) $\frac{1}{2} \left[\left(\frac{1}{3} \right)^{2} + \left(\frac{1}{3} \right)^{2} + \left(\frac{1}{3} \right)^{2} + 2 \times \left(\frac{1}{3} \right)^{2} \right]^{2}$

8) Check yield condition. $f_{n+1}^{t} = \frac{|S_{n+1}|}{\sqrt{2}} - \frac{\delta y}{\sqrt{3}}$ 9) If $f_{n+1} \leq 0$ then $\frac{1}{\sqrt{2}} + \frac{1}{\sqrt{2}} = \frac{1}{\sqrt{2}} = \frac{1}{\sqrt{2}} + \frac{1}{\sqrt{2}} = \frac{1}{\sqrt{2}} = \frac{1}{\sqrt{2}} = \frac{1}{\sqrt{2}} + \frac{1}{\sqrt{2}} = \frac{1}{$

else

i) $\Delta V = \frac{t}{A+H/3}$ -- for linear hardening

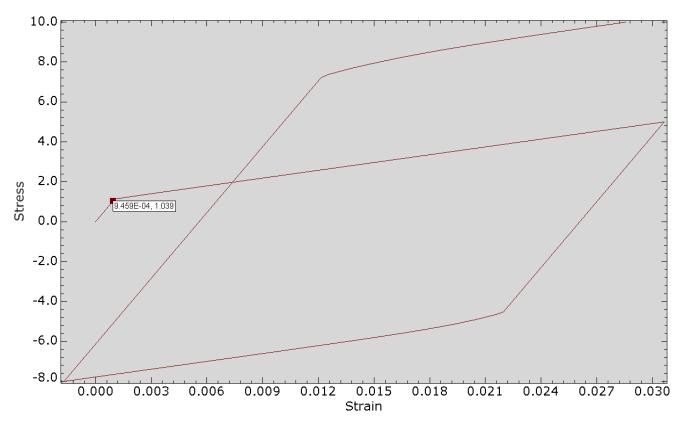
ii) $\{v_{\cdot}\}_{n+1} = \frac{1}{\sqrt{2}|\xi_{n+1}|} \{\xi_{1}\}_{1}^{3} \{\xi_{2}\}_{1}^{3}$

iii) Update plastic strain, equivalent plastic strain? 5 hress

SE = = {En3 + AP {D} n+1 dn+1 = dn + Dr 303 n+1 = 353 t - 24 AV [I] {203 n+1 + 04 313 iv) Get consistent elastic plastic temgent modulus 考n+1=1- 12 MAP : るn+1=-(- 多n+1)+1 [T],+1 = K {1} {1] + 2 u \$ n+1 ([] - \frac{1}{3} {1} {1} {1] \frac{1}{3}} -4 u 3n+1 ([] 3 23n+1) ([] 3 28n+1) T

Part a

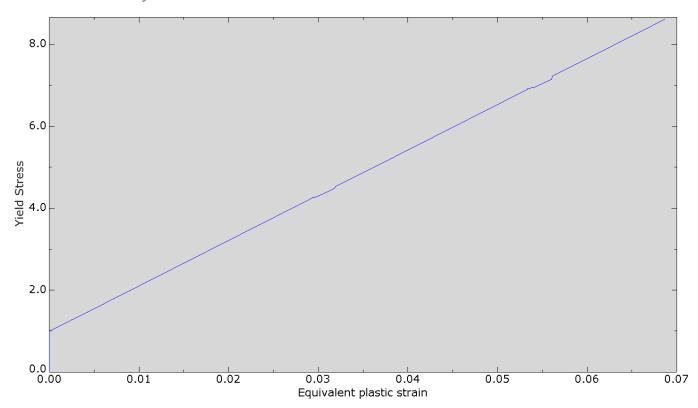
Plot of σ_{22} vs ϵ_{22}



Observations:

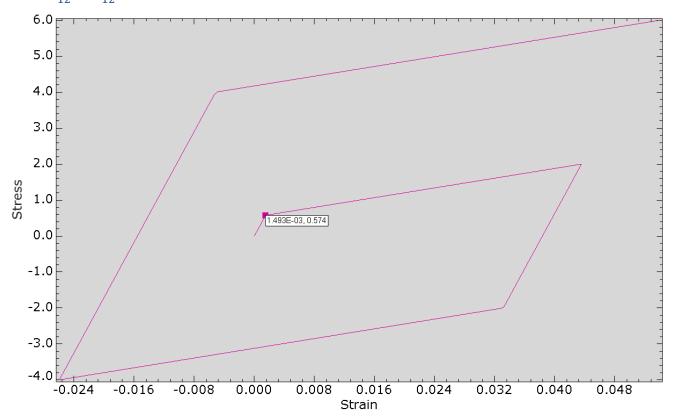
Since we are using von-mises stress criteria, we can see that the yielding starts at σ_0 for when we apply normal traction on top surface.

Plot of yield stress $\sigma_{\!\scriptscriptstyle\mathcal{Y}}$ vs equivalent plastic strain lpha



Part b

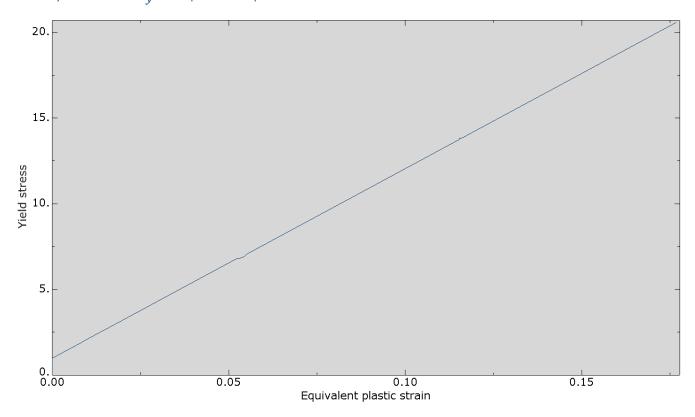
Plot of σ_{12} vs ϵ_{12}



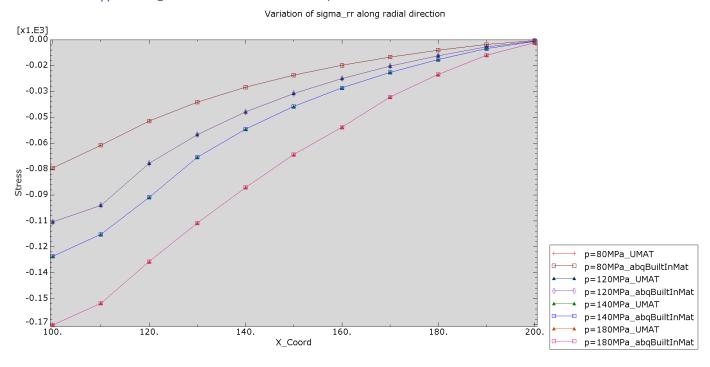
Observations:

Since we are using von-mises stress criteria, we can see that the yielding starts at $0.577*\sigma_0$ for pure shear.

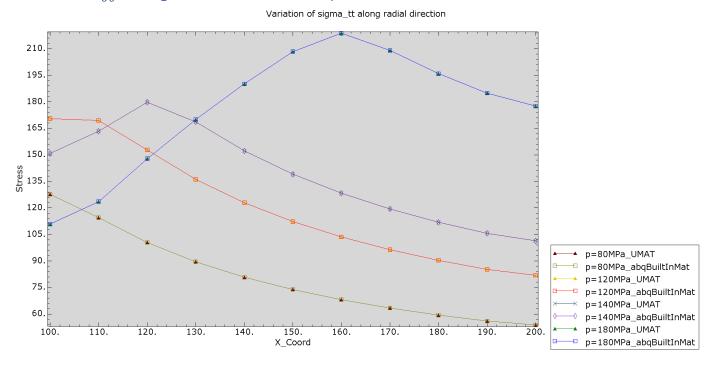
Plot of yield stress σ_y vs equivalent plastic strain lpha



Part c Variation of σ_{rr} through the thickness of the cylinder



Variation of $\sigma_{ heta heta}$ through the thickness of the cylinder



Plot of radial displacement of inner surface of the cylinder vs pressure

Radial displacement of inner surface vs pressure

