

TESTING OF TOOL

8.1 SOLVING SIMPLY SUPPORTED BEAM

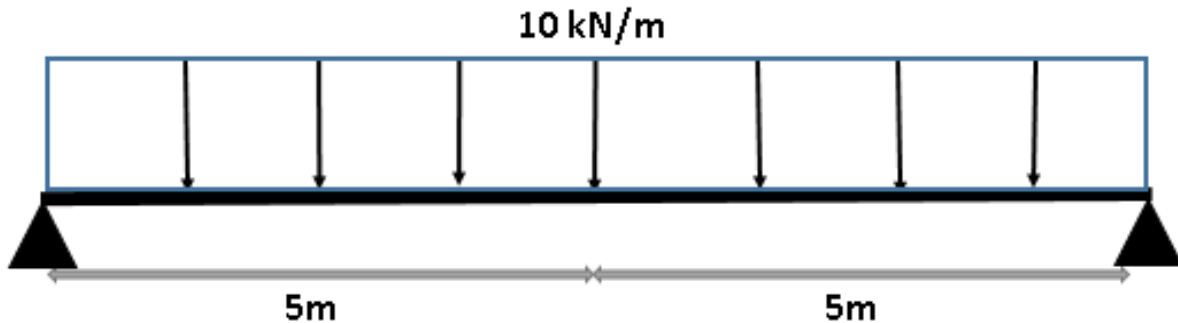


Fig 8.1 Simply supported beam

Length of the beam = 10m, and a square cross section of 0.25m*0.5m. Young's modulus is 200GPa.

$I = 2.6 \times 10^{-3} \text{ m}^4$ and $A = 0.125 \text{ m}^2$

%% Data Input Section

```
E_M=[2*10^8;2*10^8];
```

```
I_M=[2.6*10^(-3); 2.6*10^(-3)];
```

```
A_M=[0.125;0.125];
```

```
node=[0 0;5 0;10 0];
```

```
conn=[1 2;2 3];
```

```
fdof=[3 5 6 9];
```

```
f(2)=-25;f(5)=-50;f(8)=-25; % Y-forces
```

```
f(3)=-20.83;f(6)=-20.83+20.83;f(9)=20.83; % Z-moments
```

Above input snippet is based on the 3 DOF per node analogy. The output of the program is given in Fig 8.2. The Distorted geometry, BMD and the SFD are given in subsequent figures. Since, this question involves UDL, hence the shape of SFD and BMD is not correct, but the nodal values of SFD and BMD are exact.

Basic Units are kN, m and radians

Nodal Displacement

Node	X-Disp	Y-Disp	Rotation
------	--------	--------	----------

1	0.000000	0.000000	-0.000801
2	0.000000	-0.002504	-0.000000
3	0.000000	0.000000	0.000801

Member End Action (Reactions)

Node	X-Force	Y-Force	Z-Moment
------	---------	---------	----------

1	0.000000	50.000000	0.000000
2	0.000000	0.000000	0.000000
3	0.000000	50.000000	0.000000

Stress and Strain

(Preserved for future development)

Node	Direct Stress	Bending Stress	Combined Stress
1	1923.08	10000	11923.1
2	2.18629E-12	1.13687E-11	1.3555e-11
3	1923.08	10000	11923.1

END

 \vec{f}_x

Fig 8.2 Output screen for example-1

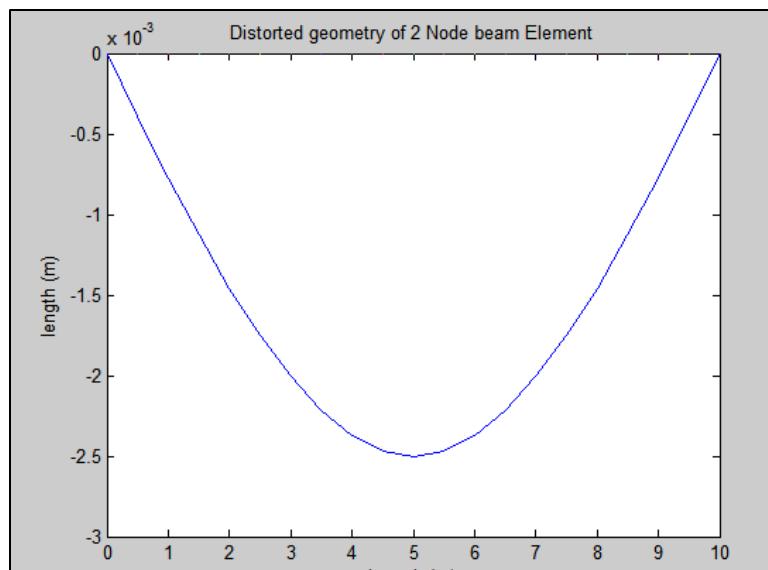


Fig 8.3 Distorted geometry of example-1

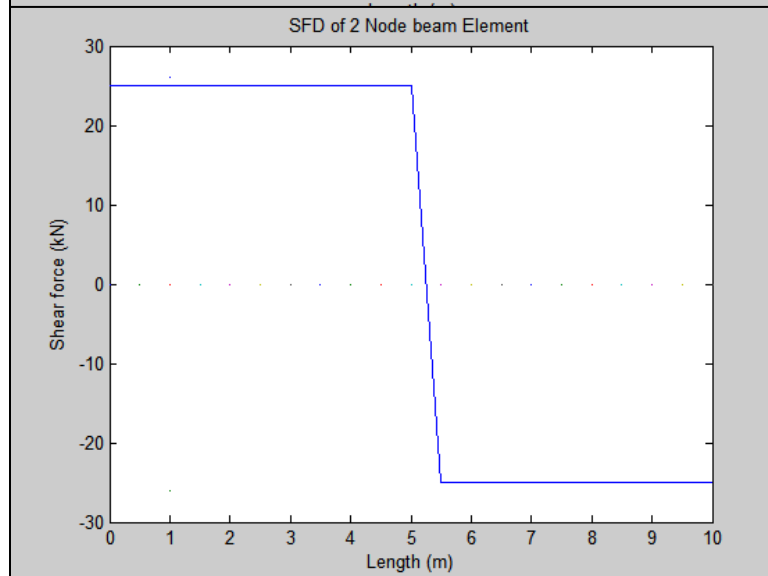


Fig 8.4 SFD of example-1

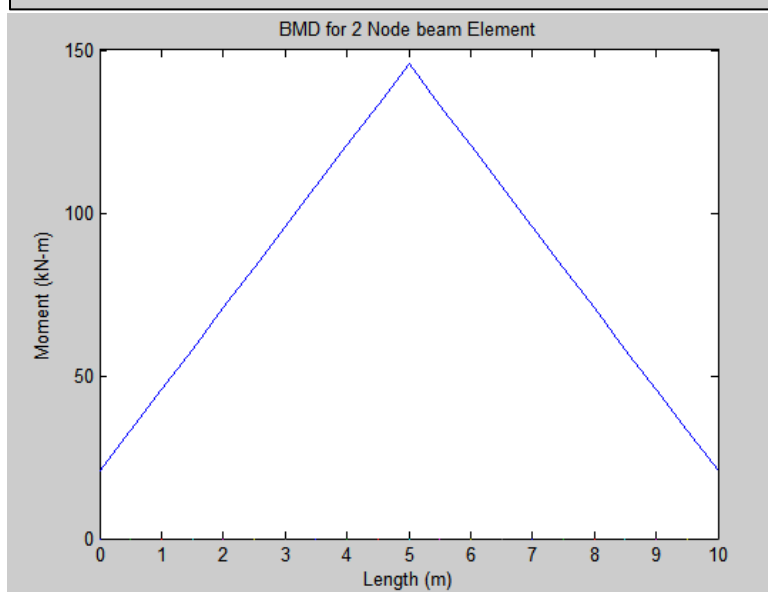


Fig 8.5 BMD of example-1

8.2 SOLVING FIXED BEAM

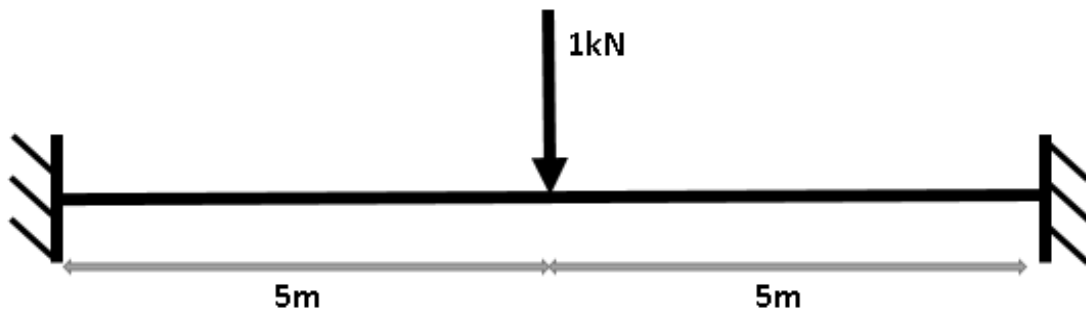


Fig 8.6 Fixed Beam

$$E = 2 \times 10^8 \text{ kN/m}^2$$

$$I = 4 \times 10^{-6} \text{ m}^4$$

%% Data Input Section

```
E_M=[2*10^8;2*10^8];
```

```
I_M=[4*10^(-6); 4*10^(-6)];
```

```
A_M=[0;0]; % Not applicable for beams
```

```
node=[0 0;5 0;10 0];
```

```
conn=[1 2;2 3];
```

```
fdof=[4 5 6];
```

```
f(5)=-1;
```

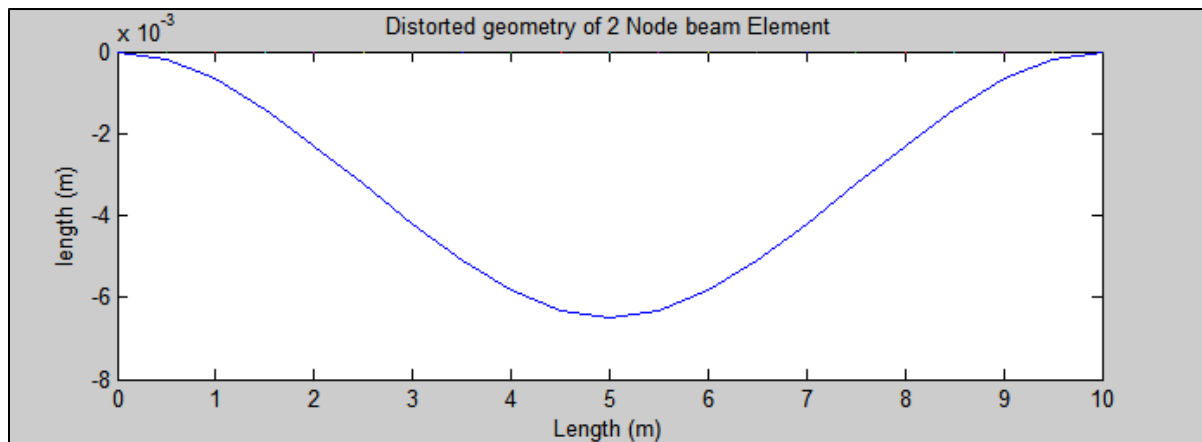


Fig 8.7 Distorted geometry of example-2

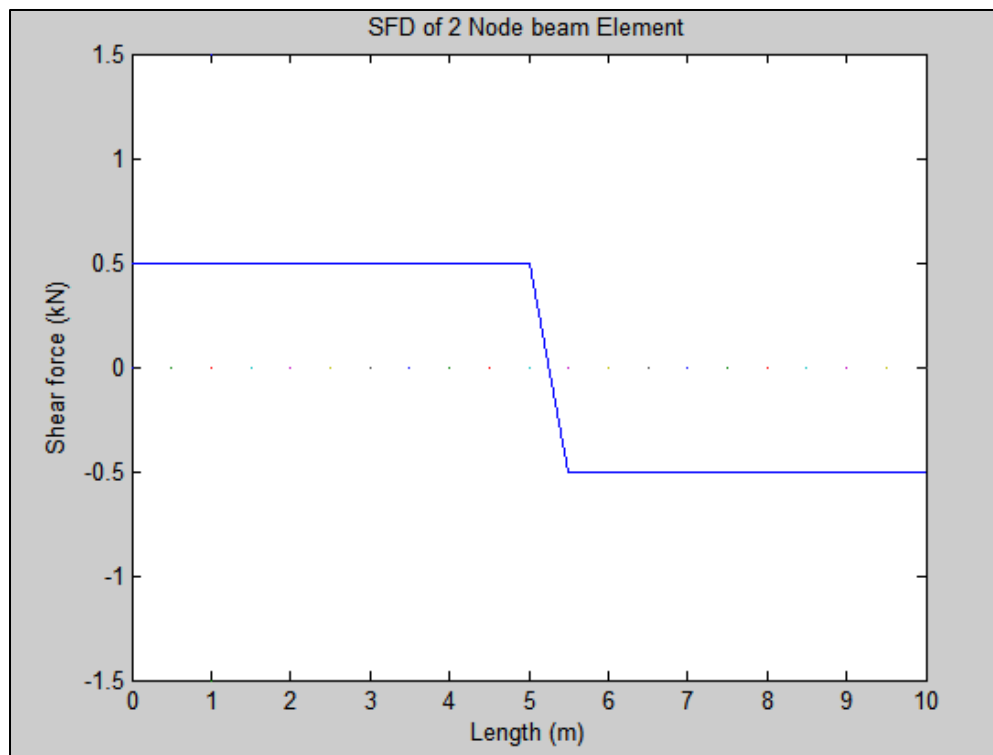


Fig 8.9 SFD for example-2

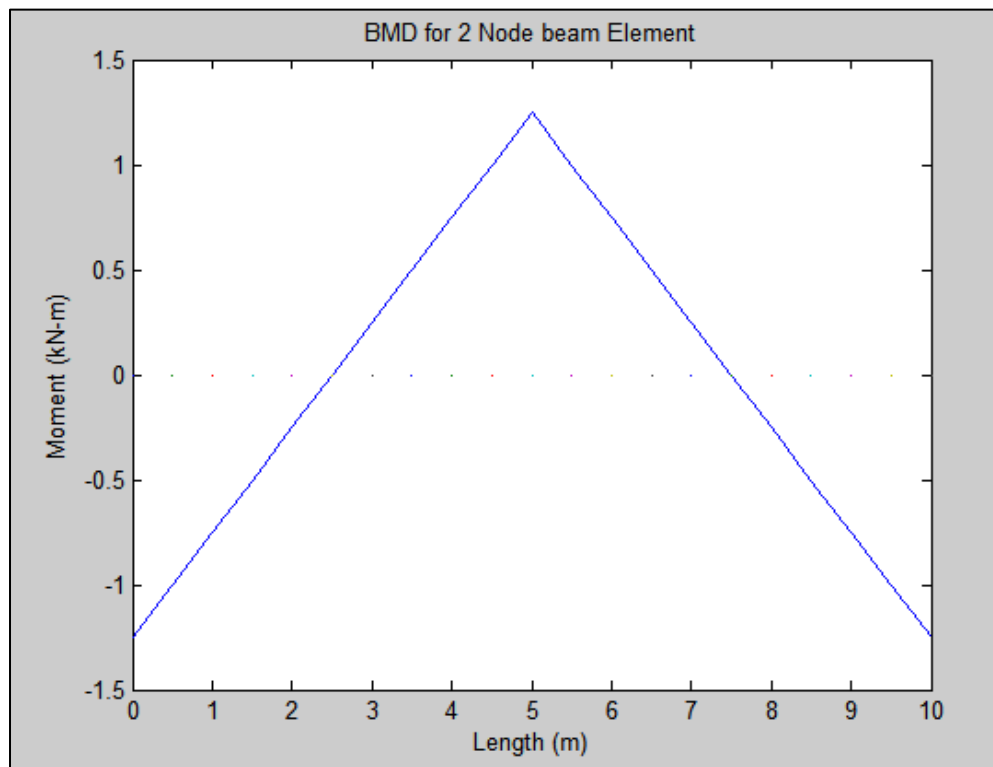


Fig 8.10 BMD for example-2

8.3 SOLVING CANTILEVER BEAM

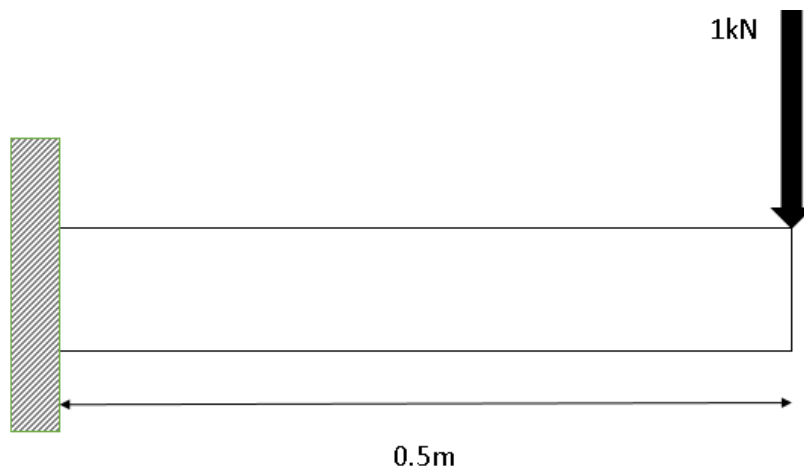


Fig 8.11 Cantilever Beam

$$E = 2 \times 10^8 \text{ kN/m}^2$$

$$I = 4 \times 10^{-6} \text{ m}^4$$

%% Data Input Section

```
E_M=[2*10^8];
```

```
I_M=[4*10^(-6)];
```

```
A_M=[0;0]; % Not applicable for beams
```

```
node=[0 0;0.5 0];
```

```
conn=[1 2];
```

```
fdof=[4 5 6];
```

```
f(5)=-1;
```

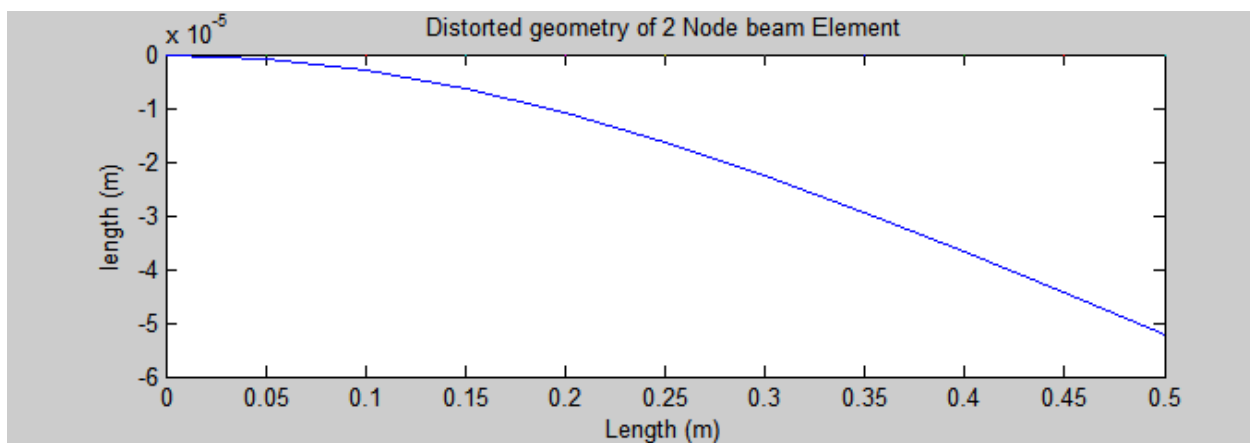


Fig 8.12 Distorted geometry of example-3

```

Command Window
Structural Analysis by Prakhar Sharma

Basic Units are kN, m and radians
-----

Nodal Displacement
-----
Node  X-Disp  Y-Disp  Rotation
1    0.000000  0.000000  0.000000
2    0.000000  -0.000052  -0.000156
-----

Member End Action (Reactions)
-----
Node  X-Force  Y-Force  Z-Moment
1    0.000000  1.000000  0.500000
2    0.000000  0.000000  0.000000
-----

Stress and Strain
(Preserved for future development)
-----
Node  Direct Stress  Bending Stress  Combined Stress
1      25000          200          25200
2    2.22045E-11    1.77636E-13    2.23821E-11
-----
END
-----

```

Fig 8.13 Output for example-3

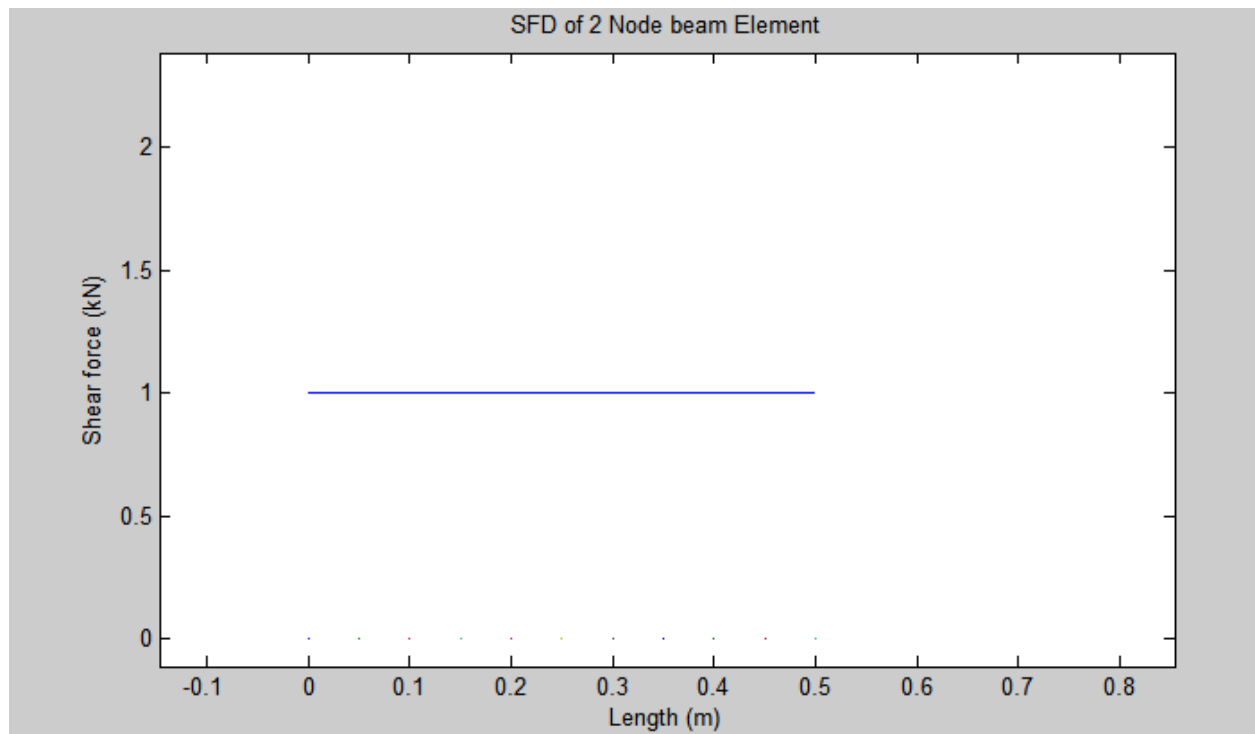


Fig 8.14 SFD for example-3

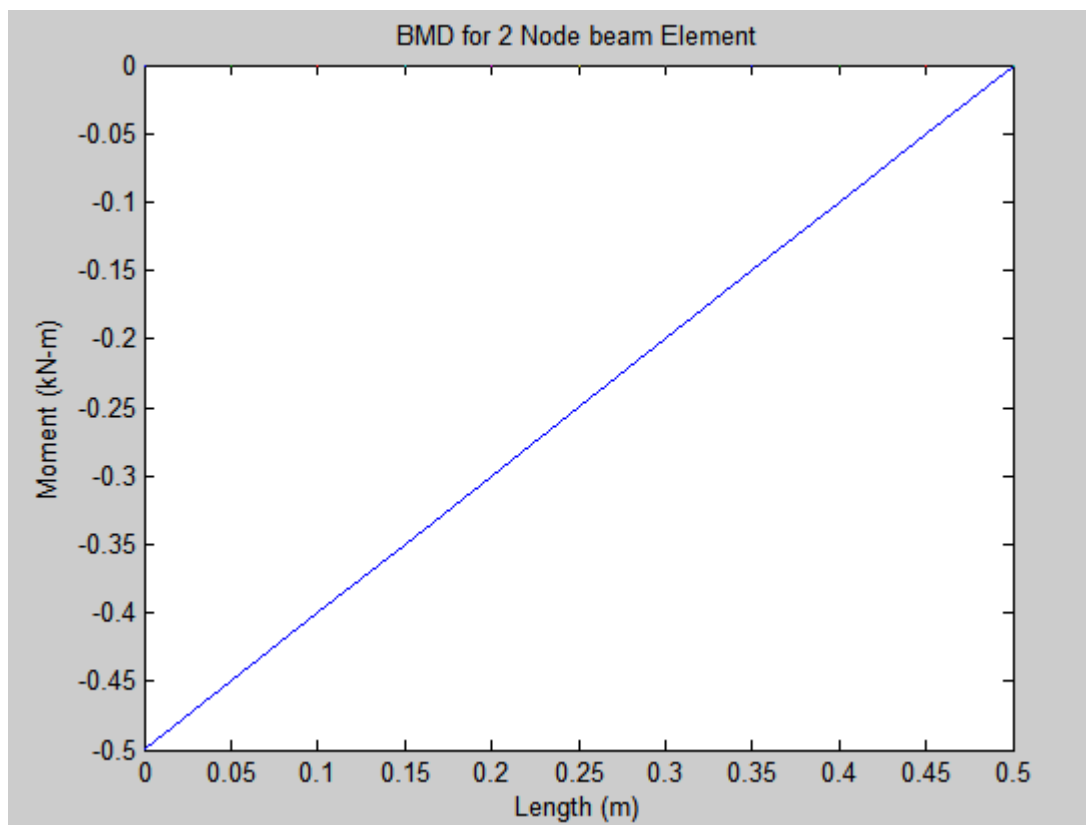


Fig 8.15 BMD for example-3

8.4 SOLVING 2D PORTAL FRAME

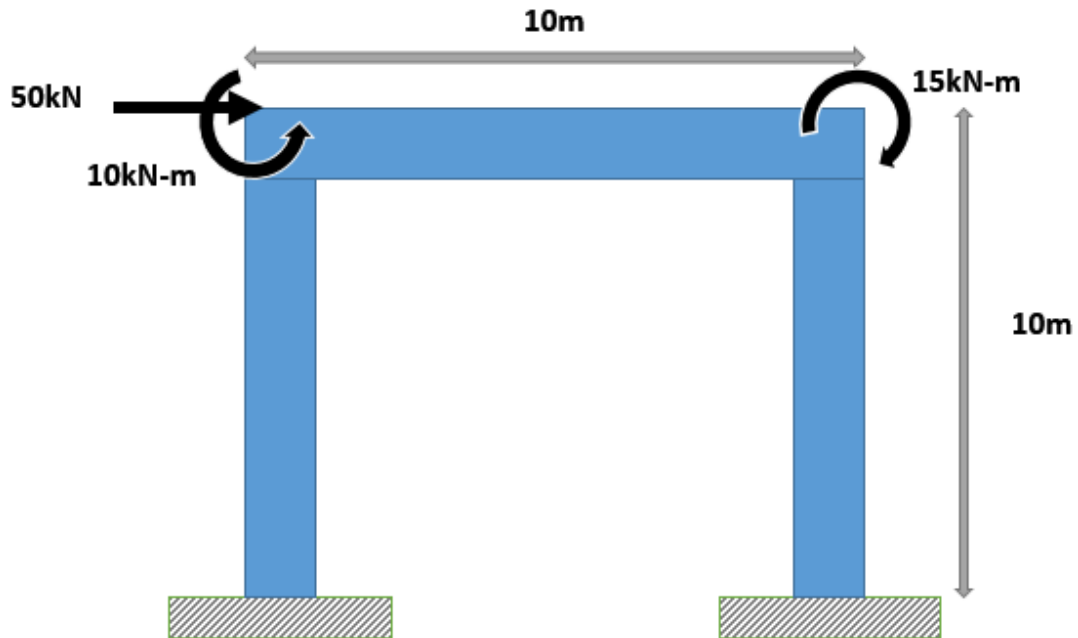


Fig 8.16 2D Frame

$E = 2 \times 10^8 \text{ kN/m}^2, 4 \times 10^8 \text{ kN/m}^2, 3 \times 10^8 \text{ kN/m}^2$

$I = 4 \times 10^{-6} \text{ m}^4, 3.56 \times 10^{-6} \text{ m}^4, 4.14 \times 10^{-6} \text{ m}^4$

$A = 0.01 \text{ m}^2, 0.02 \text{ m}^2, 0.0129 \text{ m}^2$

%% Data Input Section

```
E_M=[2*10^8,4*10^8,3*10^8];
```

```
I_M=[4*10^(-6),3.56*10^(-6),4.14*10^(-6)];
```

```
A_M=[0.01;0.02,0.0129];
```

```
node=[0 0;0 10,10 10,10 0];
```

```
conn=[1 2,2 3,3 4];
```

```
fdof=[4 5 6 7 8 9];
```

```
f(4)=50;
```

```
f(6)=10;
```

```
f(9)=-15;
```

Since, I have used trial function of insufficient order and hence the visual output will never be correct. Nevertheless, our textual output will be always true.

Command Window			
Nodal Displacement			
Node	X-Disp	Y-Disp	Rotation
1	0.000000	0.000000	0.000000
2	2.759248	0.000113	-0.078097
3	2.759214	-0.000058	-0.186036
4	0.000000	0.000000	0.000000
Member End Action (Reactions)			
Node	X-Force	Y-Force	Z-Moment
1	-22.740106	-22.564651	119.948314
2	0.000000	0.000000	0.000000
3	-0.000000	0.000000	0.000000
4	-27.259894	22.564651	159.405178
Stress and Strain (Preserved for future development)			
Node	Direct Stress	Bending Stress	Combined Stress
1	545040	4512.93	549553
2	4.29072E-11	3.55271E-13	4.32624e-11
3	8.58143E-11	7.10543E-13	8.65249e-11
4	545040	4512.93	549553
END			

Fig 8.17 Output for example-4