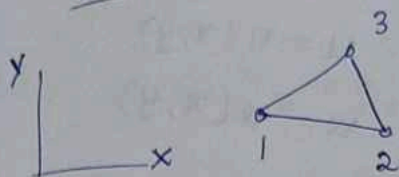


21<sup>st</sup> Oct

## Two D Vector



Each node to have 2 dofs

↳ Elasticity  
(mechanical)

- thermo - mechanical
- chemo - mechanical
- electro - mechanical
- magneto - mechanical

3 defs / node

## Elasticity

- small deformation
- linear elastic
- Cauchy stress ( $\sigma$ ), small strain ( $\epsilon$ )
- displacements ( $u$ ) —  $u$  in x-direction  
—  $v$  in y-direction

Stress components: 81 constants

Moment equilibrium  $\left\{ \begin{array}{l} 9 \text{ comp} \rightarrow 6 \text{ comp} \\ 36 \text{ constants} \end{array} \right.$

3D

$\left\{ \begin{array}{l} \sigma_x \\ \sigma_y \\ \sigma_z \\ \sigma_{xz} \\ \sigma_{zy} \\ \sigma_{yx} \\ \sigma_{yz} \\ \sigma_{xy} \end{array} \right\}$

Mtl. symmetry:  $\left\{ \begin{array}{l} \text{orthotropic} \\ \text{Transversely isotropic} \\ \text{isotropic} \end{array} \right.$  (  $\Gamma \rightarrow \downarrow$  shear  $\downarrow$   $K$  )

$$3D \rightarrow 2D$$

Consider one of the dimensions (constant) invariant

Youngs    Poiss    Bulk

plane stress:  $\sigma_z = 0$ ;  $\sigma_{zx}$ ,  $\sigma_{zy} = 0$

plane strain :  $\epsilon_z = 0$  ;  $\nu_{zx}, \nu_{zy} = 0$

bcz. of material symmetry

$$\begin{Bmatrix} \sigma_x \\ \sigma_y \\ \sigma_{xy} \end{Bmatrix}_{3 \times 1} = \underbrace{\begin{bmatrix} c_{11} & c_{12} & 0 \\ c_{21} & c_{22} & 0 \\ 0 & 0 & c_{33} \end{bmatrix}}_{\substack{\text{Elasticity matrix} \\ \text{Material matrix} \\ f(E, \nu, G, K)}} \begin{Bmatrix} \epsilon_x \\ \epsilon_y \\ \gamma_{xy} \end{Bmatrix}_{3 \times 1} \leftarrow \text{constitutive relation}$$

"strain-displacement" relation

$$\begin{Bmatrix} \epsilon_x \\ \epsilon_y \\ \gamma_{xy} \end{Bmatrix} = \begin{Bmatrix} \frac{\partial u}{\partial x} \\ \frac{\partial v}{\partial y} \\ \frac{\partial u}{\partial y} + \frac{\partial v}{\partial x} \end{Bmatrix}$$

approximate  $u, v$

$$u = \sum \phi_I(x, y) u_I \leftarrow \text{Nodal values}$$

$$v = \sum \phi_I(x, y) v_I$$

Lagrange shape functions

$$u = u(x, y)$$

$$v = v(x, y)$$

2D

Force Equilibrium

$$\begin{cases} \frac{\partial \sigma_x}{\partial x} + \frac{\partial \sigma_{xy}}{\partial y} = 0 \\ \frac{\partial \sigma_{xy}}{\partial x} + \frac{\partial \sigma_y}{\partial y} = 0 \end{cases}$$

ignore body load & inertia

(we have 2 eqs & 3 variables) cannot be solved

strain-displacement relation

constitutive relation

convert this to strain & mtl. matrix

$$\begin{Bmatrix} \sigma_x \\ \sigma_y \\ \sigma_{xy} \end{Bmatrix} \leftarrow \text{unknown}$$

convert to 2 eqns with 2 unknowns (displacements)

"displacement based formulation"

↓  
displacements becomes the primary unknowns

strain & stress: secondary unknowns

Boundary conditions — Dirichlet → on primary variable  
Neumann → secondary variable



$$\frac{\partial \sigma_x}{\partial x} + \frac{\partial \sigma_{xy}}{\partial y} = 0$$

$$\partial \Omega = \partial \Omega_N \cup \partial \Omega_D$$

$$\phi = \partial \Omega_N \cap \partial \Omega_D$$

$$\frac{\partial \sigma_{xy}}{\partial x} + \frac{\partial \sigma_y}{\partial y} = 0$$

on Dirichlet, ( $\partial \Omega_D$ ) on Neumann, ( $\partial \Omega_N$ )

$$\underline{u} = \bar{\underline{u}}$$

$$\sigma \cdot n = \hat{t} \rightarrow \text{traction}$$

line normal / surface normal (3D)  
(2D)



$$\int_{\Omega} \left( \underbrace{\frac{\partial \sigma_x}{\partial x} + \frac{\partial \sigma_{xy}}{\partial y}}_{\text{eqn 1}} + \underbrace{\frac{\partial \sigma_{xy}}{\partial x} + \frac{\partial \sigma_y}{\partial y}}_{\text{eqn 2}} \right) d\Omega = 0$$

$$u = \sum \phi_I u_I$$

$$v = \sum \phi_I v_I$$

$$g_1 = \sum \phi_I g_{1I}$$

$$g_2 = \sum \phi_I g_{2I}$$

multiply

eqn 1  $\times g_1$   $\left. \begin{matrix} g_1 \\ g_2 \end{matrix} \right\}$  arbitrary

eqn 2  $\times g_2$

$$\rightarrow \int_{\Omega} \left\{ \left( \frac{\partial \sigma_x}{\partial x} + \frac{\partial \sigma_{xy}}{\partial y} \right) g_1 + \left( \frac{\partial \sigma_{xy}}{\partial x} + \frac{\partial \sigma_y}{\partial y} \right) g_2 \right\} d\Omega = 0$$

$$\rightarrow \int_{\Omega} (g_1 \sigma_x n_x + g_1 \sigma_{xy} n_y) d\Omega - \int_{\Omega} \left( \frac{\partial g_1}{\partial x} \sigma_x + \frac{\partial g_1}{\partial y} \sigma_{xy} \right) d\Omega$$

$$+ \int_{\Omega} (g_2 \sigma_{xy} n_x + g_2 \sigma_y n_y) d\Omega - \int_{\Omega} \left( \frac{\partial g_2}{\partial x} \sigma_{xy} + \frac{\partial g_2}{\partial y} \sigma_y \right) d\Omega = 0$$

Combine,

$$\int_{\Omega} \left( \frac{\partial g_1}{\partial x} \sigma_x + \frac{\partial g_1}{\partial y} \sigma_{xy} + \frac{\partial g_2}{\partial x} \sigma_{xy} + \frac{\partial g_2}{\partial y} \sigma_y \right) d\Omega = \int_{\Omega} (g_1 n_x n_x + g_1 \sigma_{xy} n_y + g_2 \sigma_{xy} n_x + g_2 \sigma_y n_y) d\Omega$$

$$\int_{\Omega} \begin{pmatrix} \frac{\partial g_1}{\partial x} & \frac{\partial g_2}{\partial y} & \frac{\partial g_1 + \partial g_2}{\partial y} & \frac{\partial g_1 + \partial g_2}{\partial x} \end{pmatrix} \begin{Bmatrix} \sigma_x \\ \sigma_y \\ \sigma_{xy} \end{Bmatrix} d\Omega = \int_{\Omega} \begin{pmatrix} g_1 \\ g_2 \end{pmatrix}^{-1} \begin{bmatrix} \sigma_x & \sigma_{xy} \\ \sigma_{xy} & \sigma_y \end{bmatrix} \begin{bmatrix} n_x \\ n_y \end{bmatrix} d\Omega$$

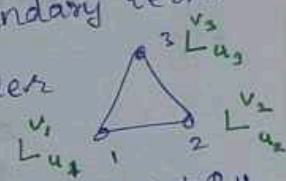
bilinear term

order can be changed

boundary term

using constitutive relation, & "strain displacement" relation

We consider



$$u = \phi_1 u_1 + \phi_2 u_2 + \phi_3 u_3$$

$$v = \phi_1 v_1 + \phi_2 v_2 + \phi_3 v_3$$

$$\int_{\Omega} \begin{pmatrix} \frac{\partial g_1}{\partial x} & \frac{\partial g_2}{\partial y} & \frac{\partial g_1 + \partial g_2}{\partial y} & \frac{\partial g_1 + \partial g_2}{\partial x} \end{pmatrix} \begin{Bmatrix} \frac{\partial u}{\partial x} \\ \frac{\partial v}{\partial y} \\ \frac{\partial u + \partial v}{\partial y} \\ \frac{\partial u + \partial v}{\partial x} \end{Bmatrix} d\Omega$$

constitutive Elasticity matrix

can be used for PL matrix formulation

$$\begin{Bmatrix} u \\ v \end{Bmatrix} = \begin{bmatrix} \phi_1 & \phi_2 & \phi_3 & 0 & 0 & 0 \\ 0 & 0 & 0 & \phi_1 & \phi_2 & \phi_3 \end{bmatrix} \begin{Bmatrix} u_1 \\ u_2 \\ u_3 \\ v_1 \\ v_2 \\ v_3 \end{Bmatrix}$$

Can be also written as

$$\begin{Bmatrix} u \\ v \end{Bmatrix} = \begin{bmatrix} \phi_1 & 0 & \phi_2 & 0 & \phi_3 & 0 \\ 0 & \phi_1 & 0 & \phi_2 & 0 & \phi_3 \end{bmatrix} \begin{Bmatrix} u_1 \\ u_2 \\ u_3 \\ v_1 \\ v_2 \\ v_3 \end{Bmatrix}$$

preferred in all commercial packages

$$\begin{Bmatrix} u_1 \\ v_1 \\ u_2 \\ v_2 \\ u_3 \\ v_3 \end{Bmatrix}$$

$$\epsilon = \begin{bmatrix} \frac{\partial \phi_1}{\partial x} & 0 & \frac{\partial \phi_2}{\partial x} & 0 & \frac{\partial \phi_3}{\partial x} & 0 \\ 0 & \frac{\partial \phi_1}{\partial y} & 0 & \frac{\partial \phi_2}{\partial y} & 0 & \frac{\partial \phi_3}{\partial y} \\ \frac{\partial \phi_1}{\partial y} & \frac{\partial \phi_1}{\partial x} & \frac{\partial \phi_2}{\partial y} & \frac{\partial \phi_2}{\partial x} & \frac{\partial \phi_3}{\partial y} & \frac{\partial \phi_3}{\partial x} \end{bmatrix} \begin{Bmatrix} u_1 \\ v_1 \\ u_2 \\ v_2 \\ u_3 \\ v_3 \end{Bmatrix}$$

B strain displacement matrix

$$\underbrace{[\leftarrow g \text{ vector} \rightarrow]}_{1 \times 6} \int_{\Omega} \underbrace{B^T}_{6 \times 3} \underbrace{\Phi}_{3 \times 3} \underbrace{B}_{3 \times 6} d\Omega \underbrace{\begin{Bmatrix} u_1 \\ v_1 \\ u_2 \\ v_2 \\ u_3 \\ v_3 \end{Bmatrix}}_{6 \times 1} = q$$

2D vector programming.

Plane stress

$$\Phi = \frac{E}{1-\nu^2} \begin{bmatrix} 1 & \nu & 0 \\ \nu & 1 & 0 \\ 0 & 0 & \frac{1-\nu}{2} \end{bmatrix}$$

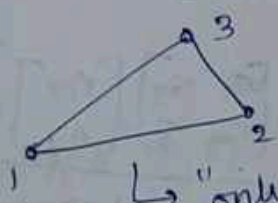
Plane strain

$$\Phi = \frac{E}{(1+\nu)(1-2\nu)} \begin{bmatrix} 1-\nu & \nu & 0 \\ \nu & 1-\nu & 0 \\ 0 & 0 & \frac{1-2\nu}{2} \end{bmatrix}$$

Elemental bilinear form

$$\int_{\Omega^e} B^T \Phi B d\Omega$$

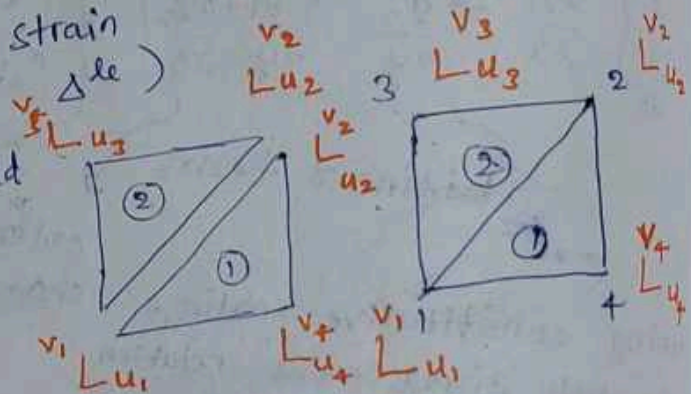
$$B = \begin{bmatrix} \frac{\partial \phi_I}{\partial x} & 0 \\ 0 & \frac{\partial \phi_I}{\partial y} \\ \frac{\partial \phi_I}{\partial y} & \frac{\partial \phi_I}{\partial x} \end{bmatrix} \quad I=1,2,3$$



3 noded  $\Delta^e$  element

CST (constant strain  $\Delta^e$ )

"only" reproduce a constant strain field





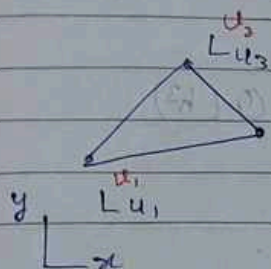
Oct 25<sup>th</sup>

$$\left. \begin{aligned} \frac{\partial \sigma_x}{\partial x} + \frac{\partial \sigma_{xy}}{\partial y} + b_x &= \rho \frac{\partial^2 u}{\partial t^2} \\ \frac{\partial \sigma_{xy}}{\partial x} + \frac{\partial \sigma_y}{\partial y} + b_y &= \rho \frac{\partial^2 v}{\partial t^2} \end{aligned} \right\}$$

$$\int_{\Omega} g_1 \left( \frac{\partial \sigma_x}{\partial x} + \frac{\partial \sigma_{xy}}{\partial y} + b_x - \rho \frac{\partial^2 u}{\partial t^2} \right) + g_2 \left( \frac{\partial \sigma_{xy}}{\partial x} + \frac{\partial \sigma_y}{\partial y} + b_y - \rho \frac{\partial^2 v}{\partial t^2} \right) d\Omega = 0$$

# custom body load

$$g_1 = \sum_I \phi_I g_{I1} ; g_2 = \sum_I \phi_I g_{I2} ; g_{I1}, g_{I2} \text{ are arbitrary}$$



$$\int_{\Omega^e} \begin{Bmatrix} \phi_1 \\ \phi_2 \\ \phi_3 \end{Bmatrix} b_x d\Omega$$

body force  
in X-direction

$$\int_{\Omega^e} \begin{Bmatrix} \phi_1 \\ \phi_2 \\ \phi_3 \end{Bmatrix} b_y d\Omega$$

body force in  
Y-direction

$$\int_{\Omega^e} \begin{bmatrix} \phi_1 & 0 & \phi_2 & 0 & \phi_3 & 0 \\ 0 & \phi_1 & 0 & \phi_2 & 0 & \phi_3 \end{bmatrix}^T \begin{bmatrix} b_x \\ b_y \end{bmatrix} d\Omega \rightarrow \text{size } 6 \times 1 \text{ vector}$$

$$\text{if } \begin{bmatrix} b_x \\ b_y \end{bmatrix} = f(x, y)$$

# Inertia

$$- \int_{\Omega} g_1 \rho \frac{\partial^2 u}{\partial t^2} d\Omega - \int_{\Omega} g_2 \rho \frac{\partial^2 v}{\partial t^2} d\Omega$$

$$u = \sum_I \phi_I(x, y) u_I(t) \quad v = \sum_I \phi_I(x, y) v_I(t)$$

$$- \int_{\Omega} \begin{bmatrix} \phi_1 & 0 & \phi_2 & 0 & \phi_3 & 0 \\ 0 & \phi_1 & 0 & \phi_2 & 0 & \phi_3 \end{bmatrix}^T \rho \begin{bmatrix} \phi_1 & 0 & \phi_2 & 0 & \phi_3 & 0 \\ 0 & \phi_1 & 0 & \phi_2 & 0 & \phi_3 \end{bmatrix} d\Omega \begin{bmatrix} \frac{\partial^2 u_1}{\partial t^2} \\ \frac{\partial^2 u_2}{\partial t^2} \\ \vdots \\ \frac{\partial^2 u_6}{\partial t^2} \end{bmatrix}$$

6x2      2x6

M: Mass matrix (6x6)

6x1

General form:-

$$M \frac{\partial^2 \underline{u}}{\partial t^2} + K \underline{u} = \underline{F} \rightarrow \text{body load: } \int_{\Omega} \phi^T b d\Omega$$

$\int_{\Omega} \phi^T \phi d\Omega$        $\int_{\Omega} B^T C B d\Omega$       BC:  $\int_{\Omega} \phi \sigma_n d\Omega$



To approximate :  $\frac{\partial^2 y}{\partial t^2}$  we use central scheme

$$f(x_0+h) = f(x_0) + h \left. \frac{\partial f}{\partial x} \right|_{x_0} + \frac{h^2}{2} \left. \frac{\partial^2 f}{\partial x^2} \right|_{x_0} + O(h^3)$$
$$+ f(x_0-h) = f(x_0) - h \left. \frac{\partial f}{\partial x} \right|_{x_0} + \frac{h^2}{2} \left. \frac{\partial^2 f}{\partial x^2} \right|_{x_0} + O(h^3)$$

$$f(x_0+h) + f(x_0-h) = 2f(x_0) + h^2 \left. \frac{\partial^2 f}{\partial x^2} \right|_{x_0} + O(h^3)$$

Transient Analysis

$$u = \underline{u}_0 e^{i\omega t} ; \quad \underline{u} = \underline{u}_0 e^{i\omega t}$$

$\downarrow \qquad \qquad \downarrow$

$$u_0(x, y) \qquad u_0(x, y)$$

Assume harmonic solution for  $\underline{u}$

$$M \ddot{u} + Ku = F$$

$$\ddot{u} = \frac{\partial^2 u}{\partial t^2}$$

$$(i\omega)^2 M \underline{u}_0 e^{i\omega t} + K \underline{u}_0 e^{i\omega t} = F$$

$$(-\omega^2 M + K) \underline{u}_0 e^{i\omega t} = F$$

Free vibration :  $F=0$  No external excitation

$$(-\omega^2 M + K) \underline{u}_0 = 0$$

Non-trivial soln.

$$\det(-\omega^2 M + K) = 0$$

eigenvalue problem

$\omega$  - frequencies - Natural frequency

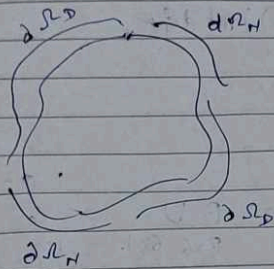
$\underline{u}_0$  - eigenvectors - mode shapes

Ex:-



Oct 28<sup>th</sup>

$$\int_{\Omega} \phi^T \rho \phi d\Omega + \int_{\Omega} \begin{bmatrix} \frac{\partial \phi_I}{\partial x} & 0 \\ 0 & \frac{\partial \phi_I}{\partial y} \\ \frac{\partial \phi_I}{\partial y} & \frac{\partial \phi_I}{\partial x} \end{bmatrix} \phi \begin{bmatrix} \frac{\partial \phi_I}{\partial x} & 0 \\ 0 & \frac{\partial \phi_I}{\partial y} \end{bmatrix} d\Omega$$



$$\frac{\partial \Omega}{N} \quad \sigma \cdot n = 0 = \begin{bmatrix} 0 \\ 0 \end{bmatrix}$$

↗ traction

$$\sigma \cdot n = \begin{bmatrix} t_x \\ t_y \end{bmatrix}$$

when  $\sigma \cdot n = 0$  } - we apply "ZERO" Neumann on the boundary  
 $q = -k \nabla T \quad q \cdot n = 0$

if  $\sigma \cdot n \neq 0$  either traction is given (or) state of stress is given

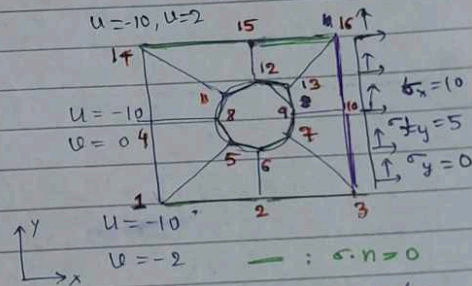
$$\begin{bmatrix} \sigma_{xx} & \sigma_{xy} \\ \sigma_{xy} & \sigma_{yy} \end{bmatrix} \cdot \begin{Bmatrix} n_x \\ n_y \end{Bmatrix}$$

$$\sigma_{xx} \cdot n_x + \sigma_{xy} \cdot n_y = t_x$$

$$\sigma_{xy} \cdot n_x + \sigma_{yy} \cdot n_y = t_y$$

EX:- Edge information

- 1 2
- 2 3
- 3 10
- 10 16
- 16 15
- 15 14
- 14 4
- 4 1



$$\int \phi \sigma \cdot n d\Omega = \text{Non zero only in } \int_3^{10} + \int_{10}^{16}$$

remaining everywhere = 0 bcz

$$\sigma \cdot n = 0$$



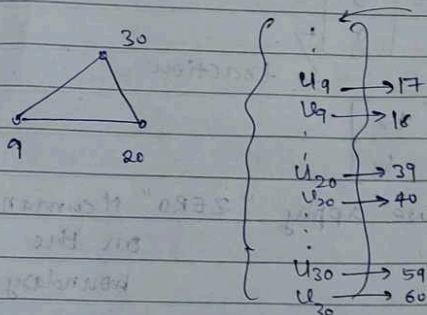
$$\int_{-1}^{+1} \phi^T \begin{bmatrix} \sigma_x & \sigma_{xy} \\ \sigma_{xy} & \sigma_y \end{bmatrix} \begin{bmatrix} n_x \\ n_y \end{bmatrix} d\Omega = \int_{-1}^{+1} \underbrace{\quad}_{4 \times 2} \begin{bmatrix} 1 & 5 \\ 5 & 0 \end{bmatrix} \begin{bmatrix} 1 \\ 0 \end{bmatrix} |J| d\xi$$

Compute contribution along x

$$\begin{Bmatrix} 5 \\ 19 \end{Bmatrix} = \int_{-1}^{+1} \begin{Bmatrix} \frac{1-\xi}{2} \\ \frac{1+\xi}{2} \end{Bmatrix} (\sigma_x n_x + \sigma_{xy} n_y) |J| d\xi$$

for the y

$$\begin{Bmatrix} 6 \\ 20 \end{Bmatrix} = \int_{-1}^{+1} \begin{Bmatrix} \frac{1-\xi}{2} \\ \frac{1+\xi}{2} \end{Bmatrix} (-\sigma_{xy} n_x + \sigma_y n_y) |J| d\xi$$



full displacement vector

$$\text{Strain} = B U$$

of element  $3 \times 1 \quad 3 \times 6 \quad 6 \times 1$

$$= \begin{bmatrix} -1 & 0 & 1 & 0 & 0 & 1 \\ 0 & -1 & 0 & 1 & 0 & 0 \\ -1 & -1 & 1 & 1 & 0 & 0 \end{bmatrix} \begin{Bmatrix} u_1 \\ u_2 \\ u_3 \\ v_1 \\ v_2 \\ v_3 \end{Bmatrix}$$

Stress

constitutive relationship

$$\sigma = \underbrace{D}_{3 \times 3} \underbrace{B}_{3 \times 6} \underbrace{U}_{6 \times 1}$$

