

# ME5204 Finite Element Analysis

## Project 2D Plane Stress Problems

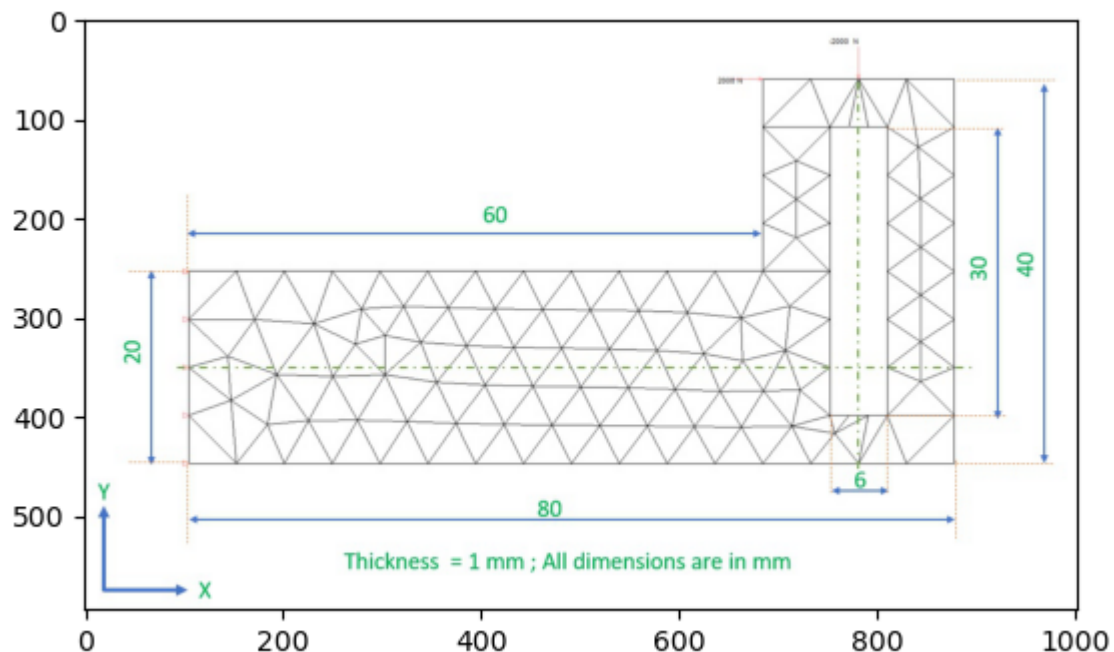
SENTHILKUMAR R (ME22M016)

```
In [1]: import math
import numpy as np
import matplotlib.pyplot as plt
from PIL import Image
```

### Problem Definition

```
In [2]: img = Image.open('problem_definition.png')
plt.imshow(img)
```

```
Out[2]: <matplotlib.image.AxesImage at 0x1fb8ea2c610>
```



### Analysis Input

```
In [3]: Num_Nodes, Num_Elems, Num_Mats, Prob_Type, Thickness = np.loadtxt("./input.txt")
Num_Nodes, Num_Elems, Num_Mats, Prob_Type = int(Num_Nodes), int(Num_Elems), int(Thickness)
```

### Nodal Co-ordinate and Array

```
In [4]: COORD = np.loadtxt("./COORD.txt").astype(np.float32)
```

```
In [5]: NCA = np.loadtxt("./NCA.txt").astype(np.int32)
#print(NCA)
```

### Material Property

```
In [6]: MAT = np.loadtxt("./MAT.txt").astype(np.float32)
        #print(MAT)
```

## Equivalent Nodal Forces

```
In [7]: LOAD_BC = np.loadtxt("./LOAD_BC.txt").astype(np.float32)
```

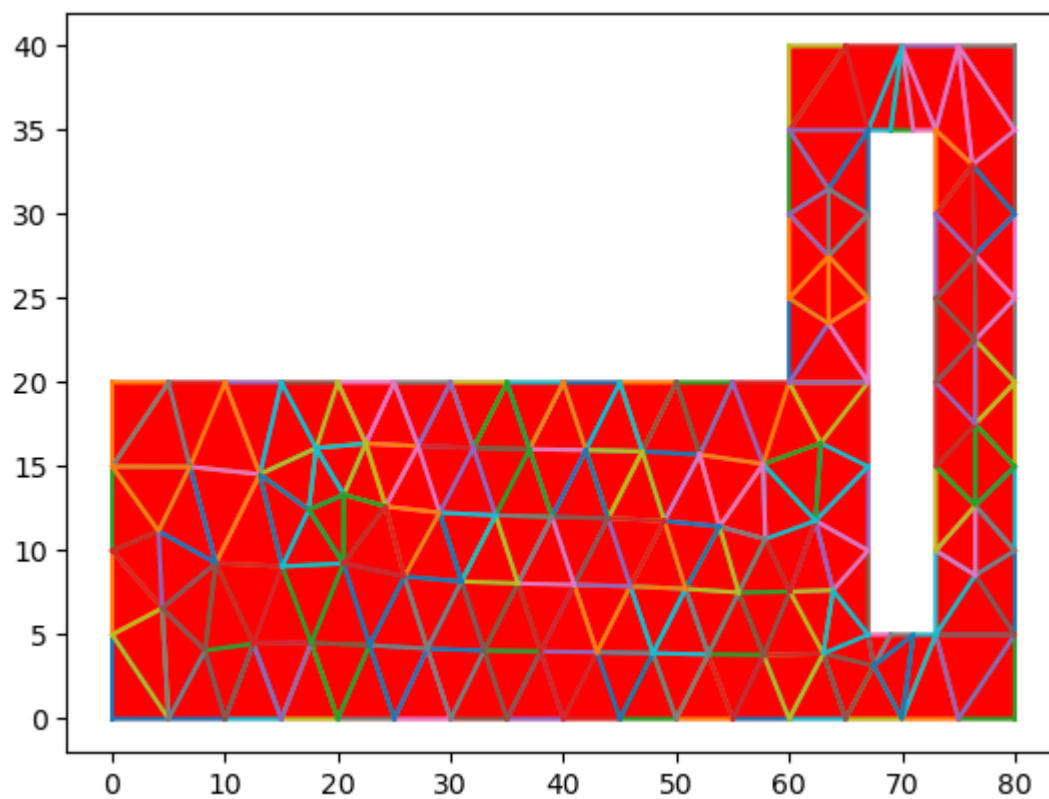
## Boundary Conditions

```
In [8]: DISP_BC = np.loadtxt("./DISP_BC.txt").astype(np.float32)
```

## Input Verification and Plot

```
In [9]: X = np.zeros([Num_Elems, 4])
        Y = np.zeros([Num_Elems, 4])
        CGX = np.zeros([Num_Elems, 1])
        CGY = np.zeros([Num_Elems, 1])

        for elem in range(1, Num_Elems+1, 1):
            N1 = NCA[elem, 3]
            N2 = NCA[elem, 2]
            N3 = NCA[elem, 1]
            X1N1 = COORD[N1, 1]
            X2N1 = COORD[N1, 2]
            X1N2 = COORD[N2, 1]
            X2N2 = COORD[N2, 2]
            X1N3 = COORD[N3, 1]
            X2N3 = COORD[N3, 2]
            Mat_Num = NCA[elem, 4]
            X = [X1N1, X1N2, X1N3, X1N1]
            Y = [X2N1, X2N2, X2N3, X2N1]
            CGX[elem - 1] = (X1N1+X1N2+X1N3)/3.0
            CGY[elem - 1] = (X2N1+X2N2+X2N3)/3.0
            plt.plot(X, Y)
        # plt.scatter(X, Y)
        #plt.text(CGX[elem-1], CGY[elem-1], str(elem), bbox = dict(facecolor = 'white',
        match NCA[elem, 4]:
            case 1:
                plt.fill(X, Y, color = 'red')
            case 2:
                plt.fill(X, Y, color = 'green')
```



## Element Stiffness matrix and Assembly of Global Stiffness matrix

```
In [10]: DOF_PN = 2
Total_DOF = Num_Nodes*DOF_PN
GSTIFF = np.zeros((Total_DOF,Total_DOF))
F = np.zeros((Total_DOF,1))

for elem in range(1, Num_Elems+1,1):
    N1 = NCA[elem,3]
    N2 = NCA[elem,2]
    N3 = NCA[elem,1]
    X1N1 = COORD[N1,1]
    X2N1 = COORD[N1,2]
    X1N2 = COORD[N2,1]
    X2N2 = COORD[N2,2]
    X1N3 = COORD[N3,1]
    X2N3 = COORD[N3,2]
    Two_Delta_matrix= np.array([[1, X1N1, X2N1],
                                [1, X1N2, X2N2],
                                [1, X1N3, X2N3]])
    Two_Delta = np.linalg.det(Two_Delta_matrix)
    Num_Nodes_PE = 3
    B = np.zeros((Num_Nodes_PE,Num_Nodes_PE*DOF_PN))
    B1 = (X2N2 - X2N3)
    B2 = (X2N3 - X2N1)
    B3 = (X2N1 - X2N2)
    G1 = (X1N3 - X1N2)
    G2 = (X1N1 - X1N3)
    G3 = (X1N2 - X1N1)
    B[0,0] = B1/Two_Delta
    B[0,2] = B2/Two_Delta
    B[0,4] = B3/Two_Delta
    B[1,1] = G1/Two_Delta
    B[1,3] = G2/Two_Delta
    B[1,5] = G3/Two_Delta
    B[2,0] = G1/Two_Delta
    B[2,1] = B1/Two_Delta
    B[2,2] = G2/Two_Delta
    B[2,3] = B2/Two_Delta
```

```

B[2,4] = G3/Two_Delta
B[2,5] = B3/Two_Delta
Two_Delta_2 = np.linalg.det(Two_Delta_matrix)
Mat_Num = NCA[elem,4]
match Mat_Num:
    case 1:
        E = MAT[Mat_Num,1]
        PR = MAT[Mat_Num,2]
    case 2:
        E = MAT[Mat_Num,1]
        PR = MAT[Mat_Num,2]
D = np.zeros((Num_Nodes_PE,Num_Nodes_PE))
match Prob_Type:
    case 21:
        CONST = E/(1-PR**2)
        D[0,0] = 1*CONST
        D[0,1] = PR*CONST
        D[0,2] = 0*CONST
        D[1,0] = PR*CONST
        D[1,1] = 1*CONST
        D[1,2] = 0*CONST
        D[2,0] = 0*CONST
        D[2,1] = 0*CONST
        D[2,2] = 0.5*(1-PR)*CONST
    case 22:
        CONST = E/((1+PR)*(1-2*PR))
        D[0,0] = (1-PR)*CONST
        D[0,1] = PR*CONST
        D[0,2] = 0*CONST
        D[1,0] = PR*CONST
        D[1,1] = (1-PR)*CONST
        D[1,2] = 0*CONST
        D[2,0] = 0*CONST
        D[2,1] = 0*CONST
        D[2,2] = 0.5*(1-2*PR)*CONST
ESTIFF = B.transpose()@D@B*Thickness*0.5*Two_Delta
CN = [2*N1-2, 2*N1-1, 2*N2-2, 2*N2-1, 2*N3-2, 2*N3-1]
CN_IDX = np.array(6*CN).reshape(6,6)
RO_IDX = CN_IDX.transpose()
GSTIFF[RO_IDX,CN_IDX] = GSTIFF[RO_IDX,CN_IDX]+ESTIFF

```

## Force Matrix

```

In [11]: Num_load = 2
F = np.zeros((Total_DOF,1))

for i in range(1, Num_load+1,1):
    LOAD_TYPE = int(LOAD_BC[i,2])
    match LOAD_TYPE:
        case 1:
            N = int(LOAD_BC[i,1])
            F[2*N-2,0] = F[2*N-2,0]+ LOAD_BC[i,3]
        case 2:
            N = int(LOAD_BC[i,1])
            F[2*N-1,0] = F[2*N-1,0]+ LOAD_BC[i,4]
        case 12:
            N = int(LOAD_BC[i,1])
            F[2*N-2,0] = F[2*N-2,0]+ LOAD_BC[i,3]
            F[2*N-1,0] = F[2*N-1,0]+ LOAD_BC[i,4]

```

Solve  $[K] \{u\} = \{F\}$  and find Displacement  $\{u\}$

```

In [12]: GSTIFFCOPY = GSTIFF.copy()
Num_Disp_BC = 5

```

```

for i in range(1, Num_Disp_BC+1, 1):
    DISP_TYPE = int(DISP_BC[i, 2])
    match DISP_TYPE:
        case 1:
            N = int(DISP_BC[i, 1])
            F[2*N-2, 0] = F[2*N-2, 0] + DISP_BC[i, 3]*10**32
            GSTIFFCOPY[2*N-2, 2*N-2] = GSTIFFCOPY[2*N-2, 2*N-2] + 10**32
        case 2:
            N = int(DISP_BC[i, 1])
            F[2*N-1, 0] = F[2*N-1, 0] + DISP_BC[i, 4]*10**32
            GSTIFFCOPY[2*N-1, 2*N-1] = GSTIFFCOPY[2*N-1, 2*N-1] + 10**32
        case 12:
            N = int(DISP_BC[i, 1])
            F[2*N-2, 0] = F[2*N-2, 0] + DISP_BC[i, 3]*10**32
            GSTIFFCOPY[2*N-2, 2*N-2] = GSTIFFCOPY[2*N-2, 2*N-2] + 10**32
            F[2*N-1, 0] = F[2*N-1, 0] + DISP_BC[i, 4]*10**32
            GSTIFFCOPY[2*N-1, 2*N-1] = GSTIFFCOPY[2*N-1, 2*N-1] + 10**32

```

```

In [13]: DISP = np.linalg.solve(GSTIFFCOPY, F)
DISP_XY = DISP.reshape(-1, 2)
DISP_XY
np.savetxt('./DISPOUT', DISP_XY)

```

## Equivalent Stress (Von-Mises Stress) Plot

```

In [14]: EQ_STRESS = np.zeros(Num_Elems+1)
for elem in range(1, Num_Elems+1, 1):
    N1 = NCA[elem, 3]
    N2 = NCA[elem, 2]
    N3 = NCA[elem, 1]
    X1N1 = COORD[N1, 1]
    X2N1 = COORD[N1, 2]
    X1N2 = COORD[N2, 1]
    X2N2 = COORD[N2, 2]
    X1N3 = COORD[N3, 1]
    X2N3 = COORD[N3, 2]
    Two_Delta_matrix = np.array([[1, X1N1, X2N1],
                                  [1, X1N2, X2N2],
                                  [1, X1N3, X2N3]])
    Two_Delta = np.linalg.det(Two_Delta_matrix)
    Num_Nodes_PE = 3
    B = np.zeros((Num_Nodes_PE, Num_Nodes_PE*DOF_PN))
    B1 = (X2N2 - X2N3)
    B2 = (X2N3 - X2N1)
    B3 = (X2N1 - X2N2)
    G1 = (X1N3 - X1N2)
    G2 = (X1N1 - X1N3)
    G3 = (X1N2 - X1N1)
    B[0, 0] = B1/Two_Delta
    B[0, 2] = B2/Two_Delta
    B[0, 4] = B3/Two_Delta
    B[1, 1] = G1/Two_Delta
    B[1, 3] = G2/Two_Delta
    B[1, 5] = G3/Two_Delta
    B[2, 0] = G1/Two_Delta
    B[2, 1] = B1/Two_Delta
    B[2, 2] = G2/Two_Delta
    B[2, 3] = B2/Two_Delta
    B[2, 4] = G3/Two_Delta
    B[2, 5] = B3/Two_Delta
    Two_Delta_2 = np.linalg.det(Two_Delta_matrix)
    Mat_Num = NCA[elem, 4]
    match Mat_Num:
        case 1:
            E = MAT[Mat_Num, 1]

```

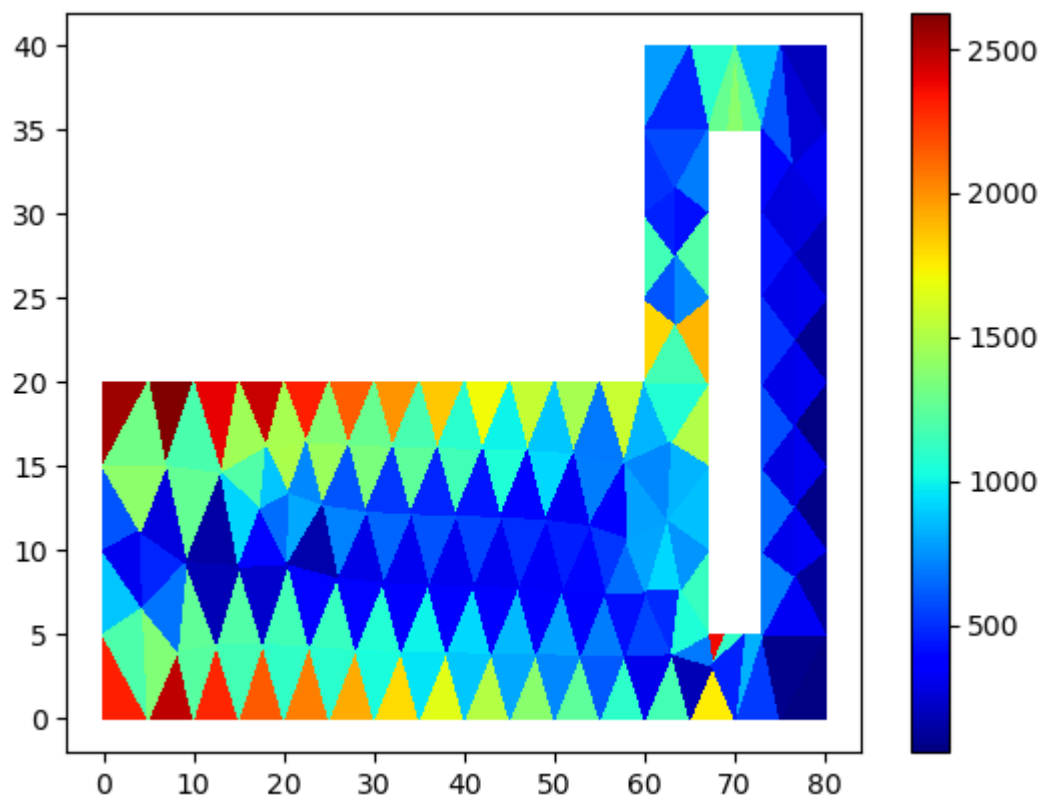
```

PR = MAT[Mat_Num,2]
case 2:
    E = MAT[Mat_Num,1]
    PR = MAT[Mat_Num,2]
D = np.zeros((Num_Nodes_PE, Num_Nodes_PE))
match Prob_Type:
    case 21:
        CONST = E/(1-PR**2)
        D[0,0] = 1*CONST
        D[0,1] = PR*CONST
        D[0,2] = 0*CONST
        D[1,0] = PR*CONST
        D[1,1] = 1*CONST
        D[1,2] = 0*CONST
        D[2,0] = 0*CONST
        D[2,1] = 0*CONST
        D[2,2] = 0.5*(1-PR)*CONST
    case 22:
        CONST = E/((1+PR)*(1-2*PR))
        D[0,0] = (1-PR)*CONST
        D[0,1] = PR*CONST
        D[0,2] = 0*CONST
        D[1,0] = PR*CONST
        D[1,1] = (1-PR)*CONST
        D[1,2] = 0*CONST
        D[2,0] = 0*CONST
        D[2,1] = 0*CONST
        D[2,2] = 0.5*(1-2*PR)*CONST
CN = [2*N1-2, 2*N1-1, 2*N2-2, 2*N2-1, 2*N3-2, 2*N3-1]
u = DISP[CN]
STRAIN = B@u
STRESS = D@STRAIN
STRESS = STRESS.reshape(-1,3)
STRESS_XX = STRESS[:,0]
STRESS_YY = STRESS[:,1]
STRESS_XY = STRESS[:,2]
V = np.sqrt((STRESS_XX**2-(STRESS_XX*STRESS_YY))+(STRESS_YY**2)+(3*(STRESS_XY**2)))

EQ_STRESS[elem] = V

XCOORD = COORD[1:, 1]
YCOORD = COORD[1:, 2]
ELEM_CON = NCA[1:,1:-1]-1
VonMises = EQ_STRESS[1:]
plt.tripcolor(XCOORD, YCOORD, ELEM_CON, VonMises, cmap='jet')
plt.colorbar()
plt.show()

```



Deformed shape Vs Un-deformed Shape Plot

```
In [15]: COORD_NEW = COORD[1:, [1,2]] + DISP_XY
for elem in range(1, Num_Elems+1,1):
    N1 = NCA[elem,3]
    N2 = NCA[elem,2]
    N3 = NCA[elem,1]

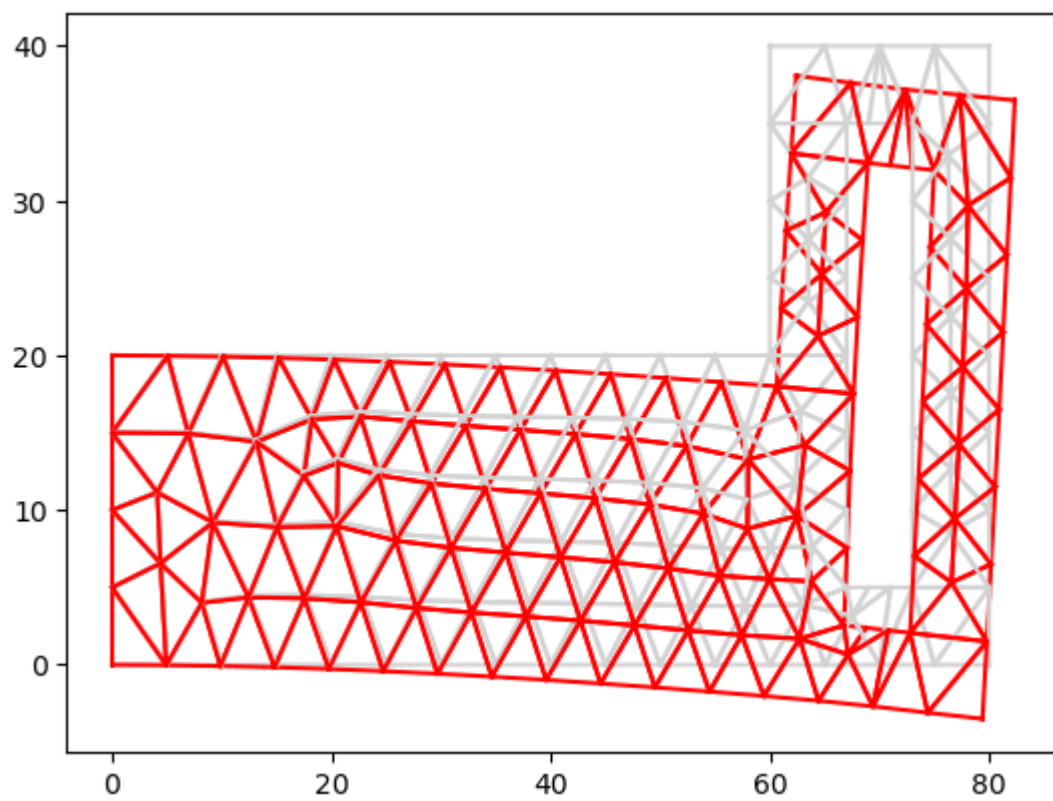
    X1N1 = COORD[N1,1]
    X2N1 = COORD[N1,2]
    X1N2 = COORD[N2,1]
    X2N2 = COORD[N2,2]
    X1N3 = COORD[N3,1]
    X2N3 = COORD[N3,2]

    X = [X1N1,X1N2,X1N3,X1N1]
    Y = [X2N1,X2N2,X2N3,X2N1]

    plt.plot(X, Y, color='lightgrey')

    X1N1_NEW = COORD_NEW[N1-1,0]
    X2N1_NEW = COORD_NEW[N1-1,1]
    X1N2_NEW = COORD_NEW[N2-1,0]
    X2N2_NEW = COORD_NEW[N2-1,1]
    X1N3_NEW = COORD_NEW[N3-1,0]
    X2N3_NEW = COORD_NEW[N3-1,1]
    Mat_Num = NCA[elem, 4]
    X = [X1N1_NEW,X1N2_NEW,X1N3_NEW,X1N1_NEW]
    Y = [X2N1_NEW,X2N2_NEW,X2N3_NEW,X2N1_NEW]

    plt.plot(X, Y, color='red')
```



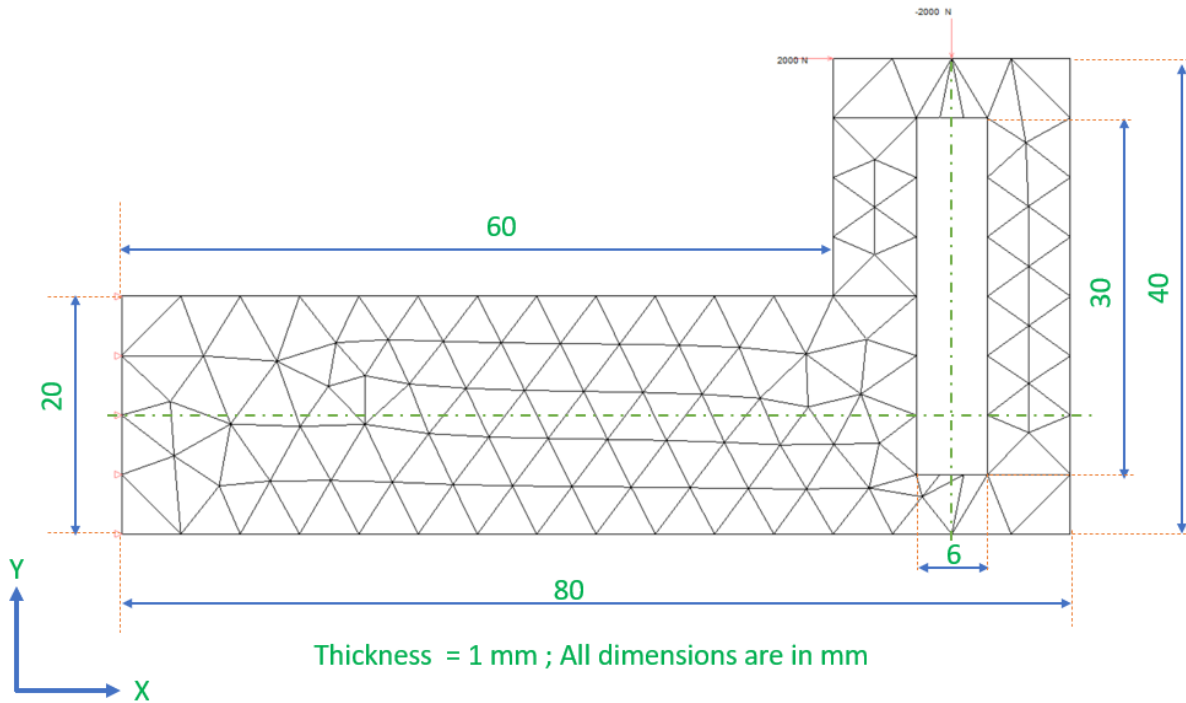
In [ ]:



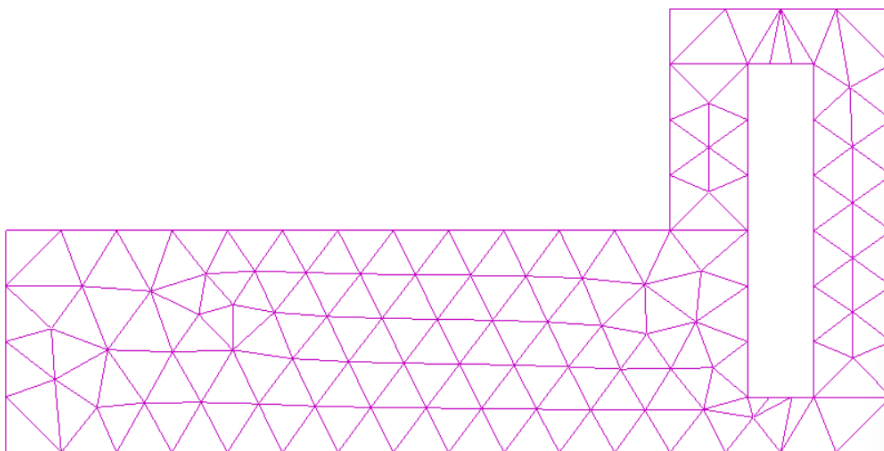
## Analysis By VisualFEA

### Problem Definition

#### Plane Stress Analysis:



### Property



Property	
Assign	New Delete
1	P1
<input type="checkbox"/> Property color	
Plane / Surface	?
Linear	
<input checked="" type="radio"/> Isotropic	
<input type="radio"/> Orthotropic	
<input type="checkbox"/> Undrained	
E	200000
ν	0.2
t	1
ω <sub>0</sub>	1e-06
α	1e-05

- Element Type: 3 Node Tria (CST)
- No. of Elements: 184
- No. of Nodes: 125

### Boundary:

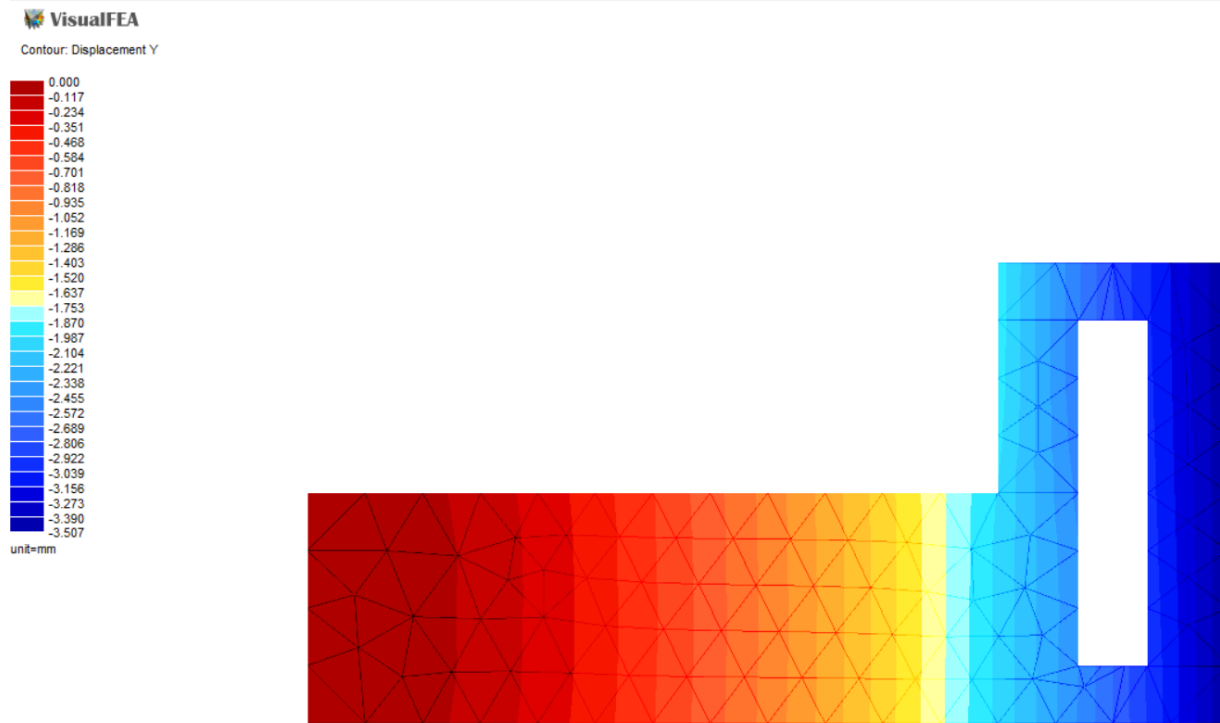
- Nodes in left most surface has fixed.

### Load Condition:

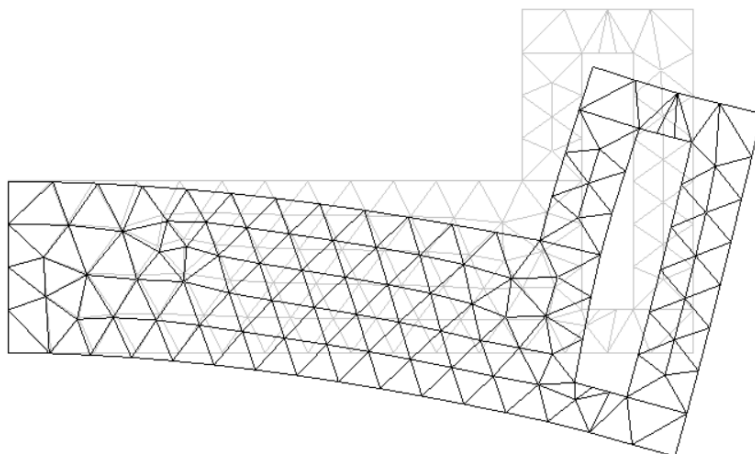
- P in X direction as per above figure: 2000 N
- P in -Y direction as per above figure: -2000 N

### Results:

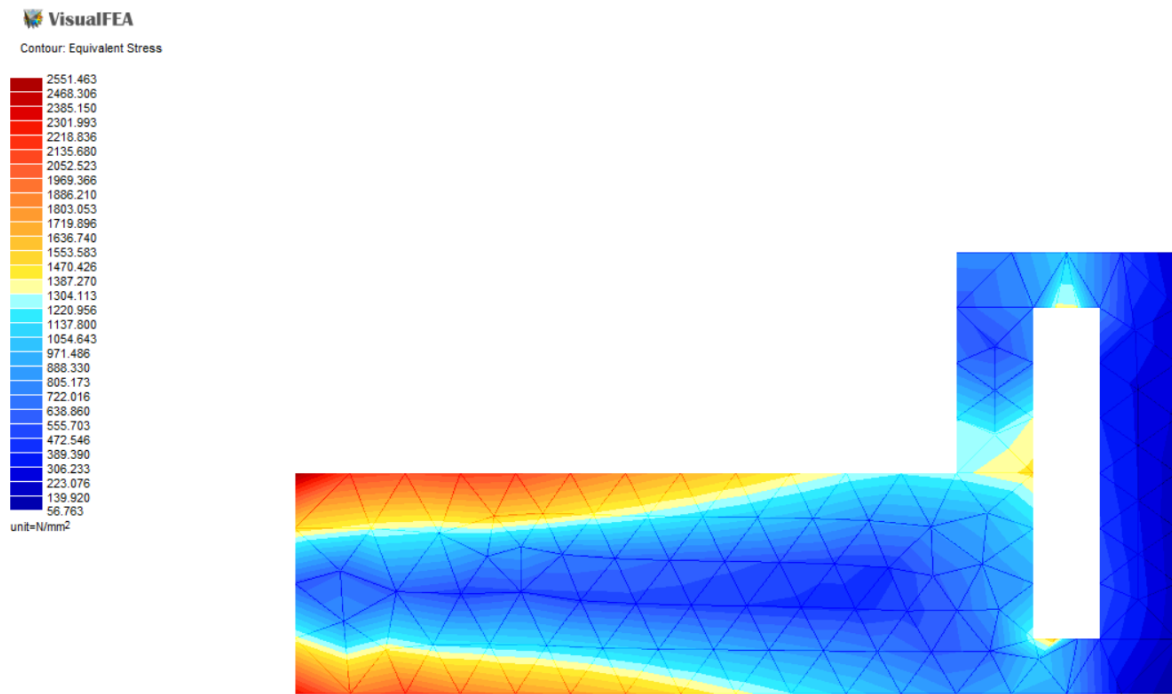
#### Displacements in Y



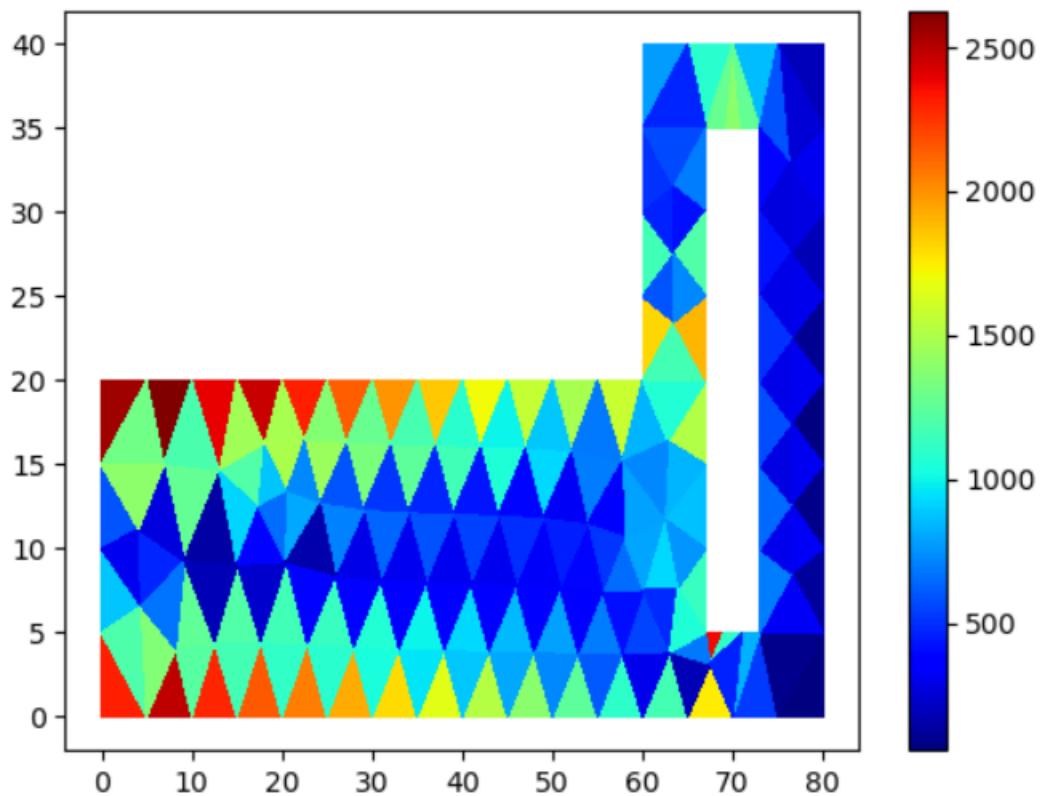
#### Deformed Shape



## Equivalent Stress:

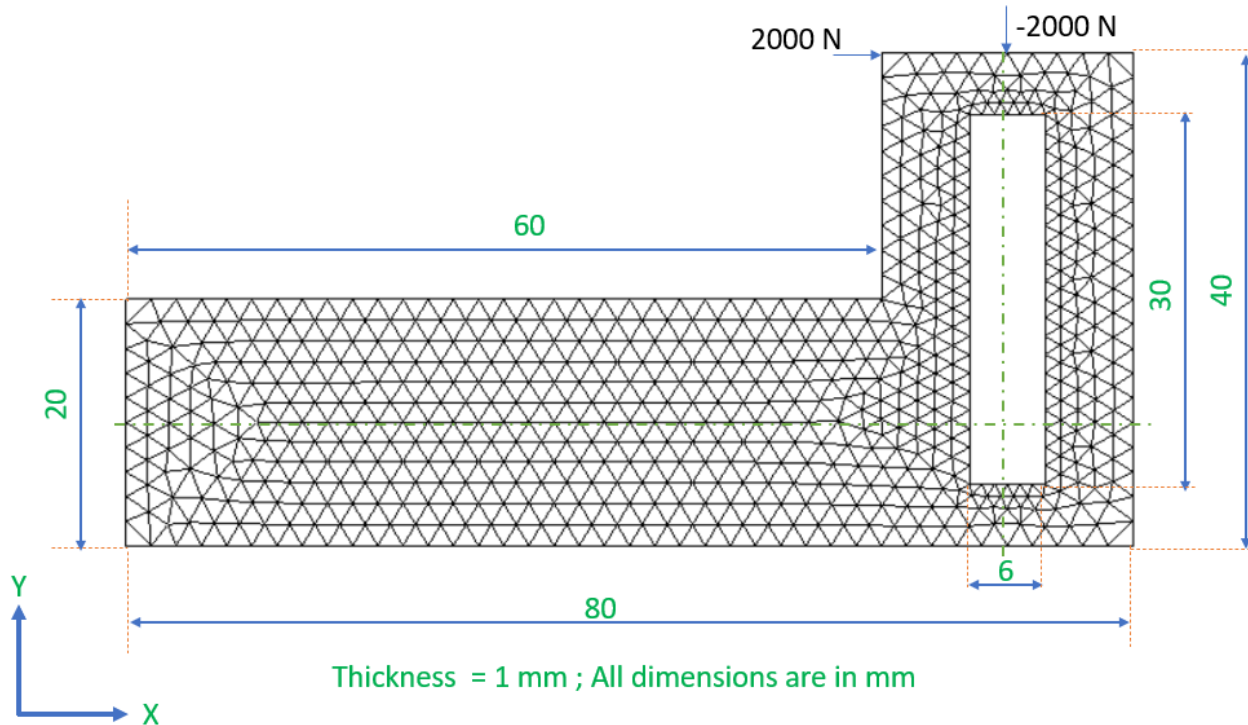


## Equivalent Stress by Python code:

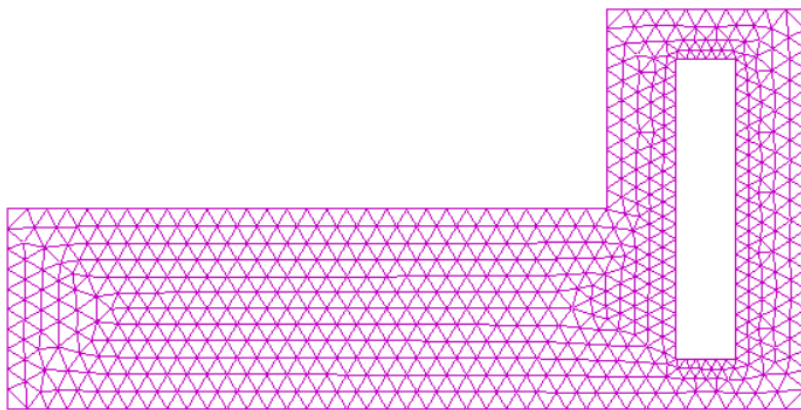


## Problem Definition

Plane Stress Analysis: (geometry same as earlier one)



## Property



Property

Assign New Delete

1 P1

☐ Property color

Plane / Surface ?

Linear

☒ Isotropic

☐ Orthotropic

☐ Undrained

E	200000
$\nu$	0.2
t	1
$\omega_0$	1e-06
$\alpha$	1e-05

- Element Type: 3 Node Tria (CST)
- No. of Elements: 708
- No. of Nodes: 1244

## Boundary:

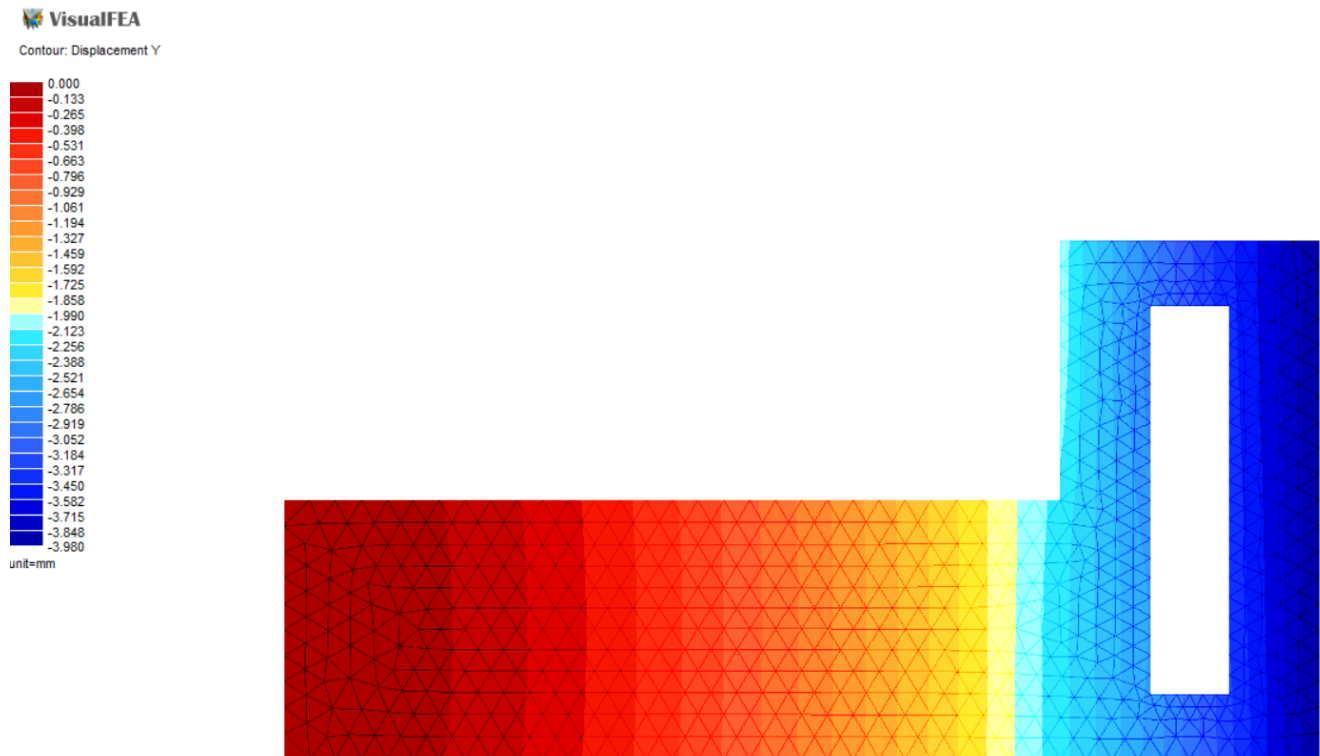
Nodes in left most surface has fixed.

## Load Condition:

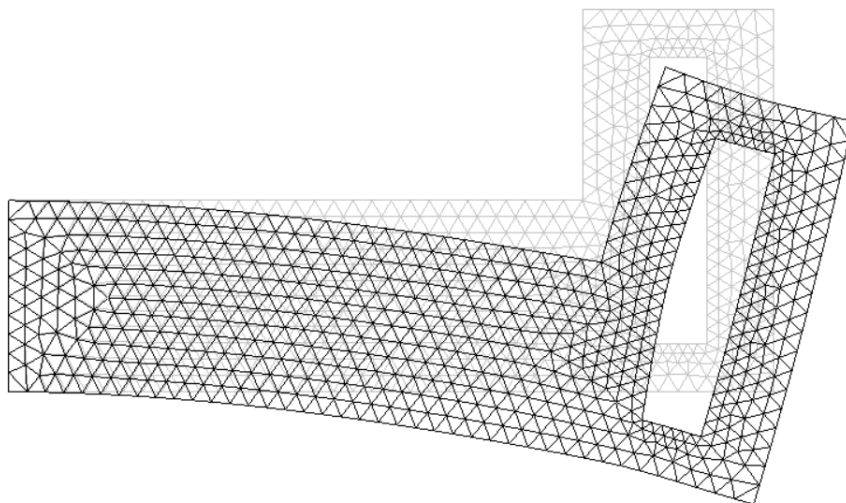
- P in X direction as per above figure: 2000 N
- P in -Y direction as per above figure: -2000 N

## Results:

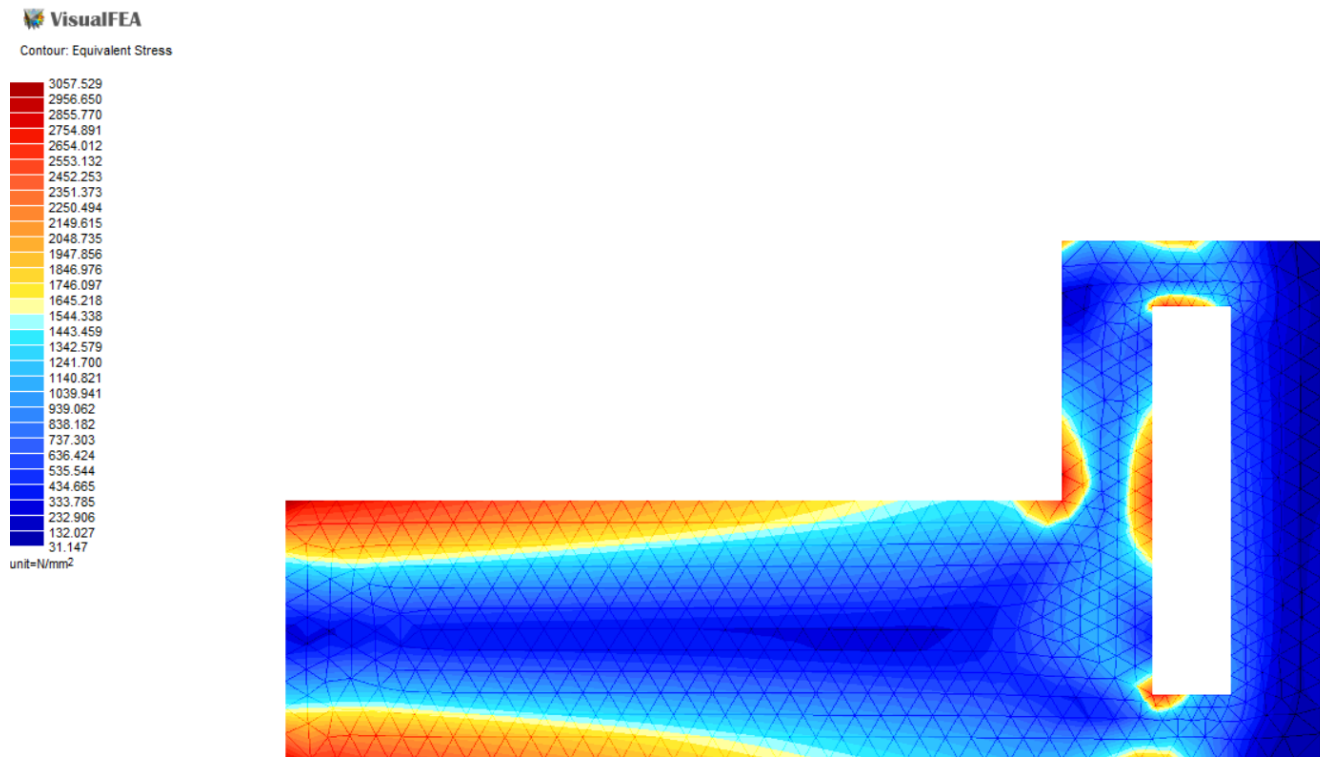
### Displacements in Y



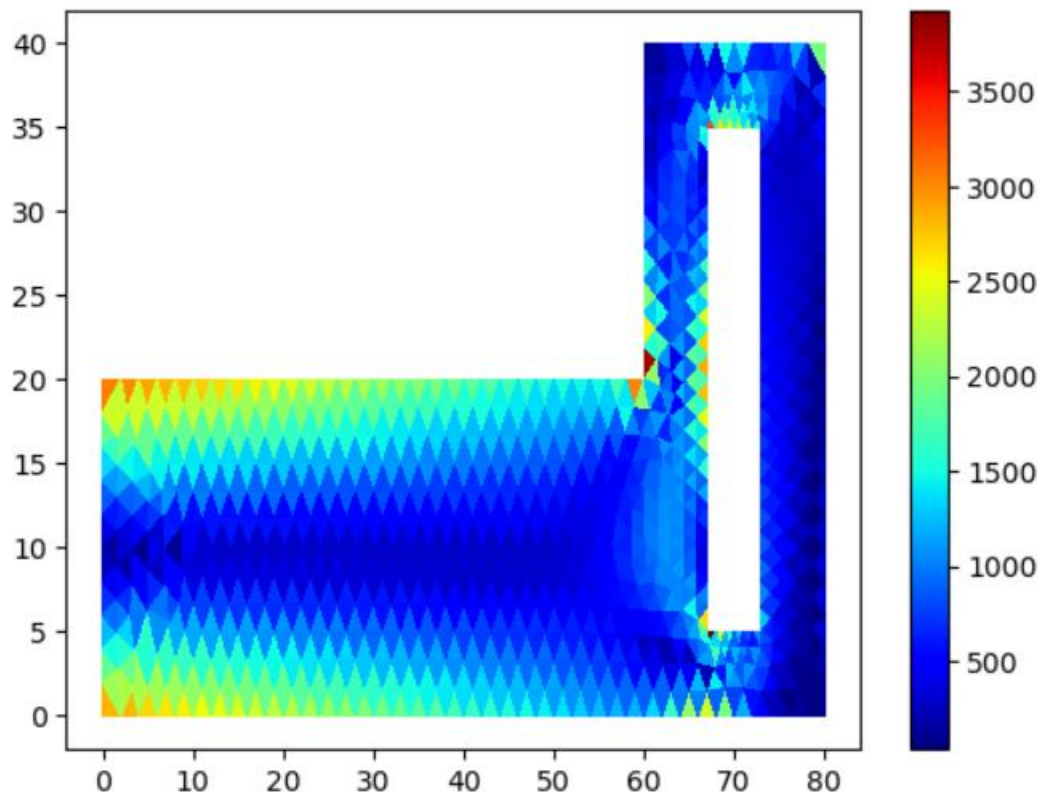
### Deformed Shape



## Equivalent Stress:



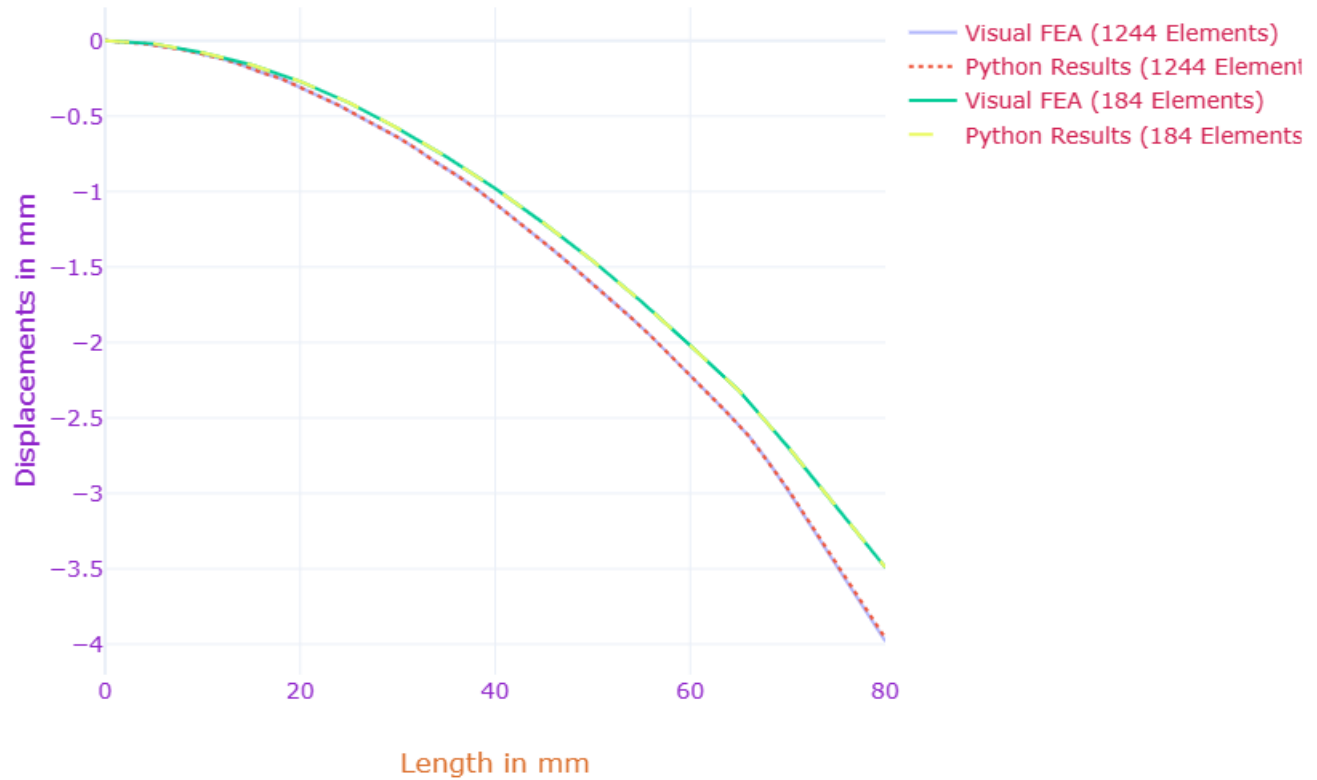
## Equivalent Stress by Python code:



## Convergence study with respect to No. of Elements

Displacements in Y direction has taken for Nodes in co-ordinate in  $y = 0$ .

### Convergence with respect to Number of Elemets



For 184 Elements

X -Coordinates	0	5	10	15	20	25	30	35	40	45	50	55	60	65	70	75	80
Y Displacement by VisualFEA	0	-0.02	-0.08	-0.16	-0.27	-0.41	-0.58	-0.77	-0.98	-1.21	-1.46	-1.73	-2.02	-2.32	-2.69	-3.09	-3.49
Y Displacement by Python Code	0	-0.02	-0.08	-0.16	-0.27	-0.41	-0.58	-0.77	-0.98	-1.21	-1.46	-1.73	-2.02	-2.32	-2.69	-3.09	-3.49

For 1244 Elements

X -Coordinates	0	2	4	6	8	10	12	14	16	18	20	22	24	26	28	30	32	34	36	38	40
Y Displacement by VisualFEA	0	-0.01	-0.02	-0.04	-0.06	-0.09	-0.12	-0.16	-0.21	-0.25	-0.31	-0.37	-0.43	-0.5	-0.57	-0.64	-0.72	-0.81	-0.89	-0.98	-1.08
Y Displacement by Python Code	0	-0.01	-0.02	-0.04	-0.06	-0.09	-0.12	-0.16	-0.21	-0.25	-0.31	-0.37	-0.43	-0.5	-0.57	-0.64	-0.72	-0.81	-0.89	-0.98	-1.08

X -Coordinates	42	44	46	48	50	52	54	56	58	60	62	64	66	68	70	72	74	76	78	80
Y Displacement by VisualFEA	-1.18	-1.28	-1.39	-1.5	-1.61	-1.72	-1.84	-1.96	-2.09	-2.21	-2.34	-2.48	-2.62	-2.79	-2.98	-3.18	-3.38	-3.58	-3.78	-3.98
Y Displacement by Python Code	-1.18	-1.28	-1.39	-1.5	-1.61	-1.72	-1.84	-1.96	-2.09	-2.21	-2.34	-2.48	-2.62	-2.79	-2.97	-3.17	-3.37	-3.57	-3.76	-3.96

### **Conclusion:**

it can be concluded that the **Python code** and **VisualFEA** software produce identical results when subjected to the same property and boundary conditions. Furthermore, it was found that increasing the number of elements improved the accuracy of the results. These findings demonstrate the effectiveness and **reliability of both the Python code and VisualFEA software** in analyzing structures. Therefore, Python code can be considered suitable for use and it is recommended to continue exploring the capabilities of these tools in future research to further enhance their accuracy and efficiency.