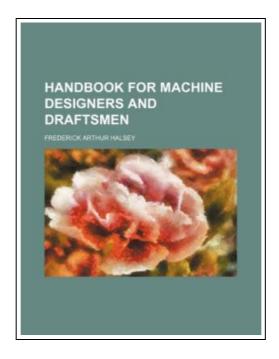
Handbook for Machine Designers and Draftsmen



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(Cordie Hauck DVM)

HANDBOOK FOR MACHINE DESIGNERS AND DRAFTSMEN



Rarebooksclub.com, United States, 2012. Paperback. Book Condition: New. 246 x 189 mm. Language: English . Brand New Book ***** Print on Demand ******This historic book may have numerous typos and missing text. Purchasers can download a free scanned copy of the original book (without typos) from the publisher. Not indexed. Not illustrated. 1913 Excerpt: the square of the tV outside diameter in in., we get equal.347. Finding this value on the scale at the lower margin of Fig. 2 we trace vertically until the line XX is reached, then horizontally toward the right and read 1525 lbs. per sq. in. as the probable collapsing pressure required. While this value is for a 20-ft. length of tube, as in the preceding chart, it may be used without substantial error for any length greater than about six diameters, or in this case 3 ft., between joints tending to hold the tube to a circular form. Scale for Y Scale for Y Thickness Outside Diameter Plain End Weight Diameter Thickness Outside Diameter Plain End Weighty Diameter 1 Fig. L--Plotted in terms of thickness. Fig. 2.--Plotted in terms of weight. Fics. 1 and 2.--The strength of Bessemer steel tubes against collapsing pressure. Professor Stewart s paper contains the following observations: The apparent fiber stress under which the different tubes failed varied from about 7000 lbs. for the relatively thinnest to 35,000 lbs. per sq. in. for the relatively thickest walls. Since the average yield point of the material was 37,000 and the tensile strength 58,000 lbs. per sq. in., it would appear that the strength of a tube subjected to a fluid collapsing pressure is not dependent alone upon either the elastic limit or ultimate strength of the material constituting it. The experiments show that the element of greatest weakness in a commercial lap-welded...



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