

UNIVERSITY OF TORONTO
FACULTY OF APPLIED SCIENCE AND ENGINEERING
DEPARTMENT OF MECHANICAL ENGINEERING

First Year - MEC100S

DYNAMICS

Final Examination

Date: April 24, 2002

Time: 2:00-4:30

Instructions:

1. Answer all the questions.
2. 5 questions only and each question is worth 20%.
3. Only non-programmable calculators are allowed.

1. The 2 kg collar is released from rest at *A* and slides down the inclined fixed rod in the vertical plane, Fig.1. The coefficient of kinetic friction is 0.4. Calculate (a) the velocity *v* of the collar as it strikes the spring and (b) the maximum deflection *x* of the spring.

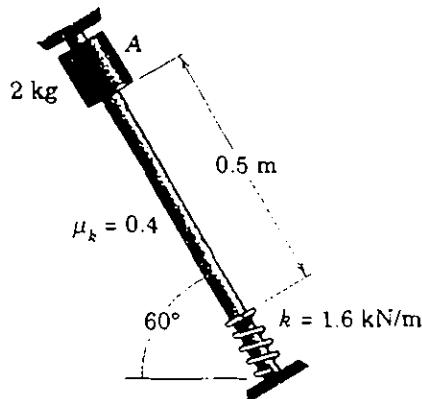


Figure 1

2. Motion of link *ABC* is controlled by the horizontal movement of the piston rod of the hydraulic cylinder *D* and by the vertical guide for the pinned slider at *B*, Fig.2. For the instant when $\theta = 45^\circ$, the piston rod is retracting at the constant rate $v_c = 180 \text{ mm/s}$. For this instant determine (a) angular acceleration of bar AC, and (b) the acceleration of point *A*.

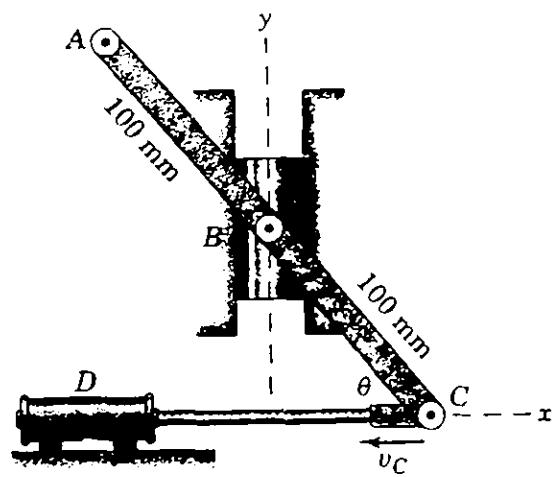


Figure 2

3. The semicircular disk having a mass of 10 kg , Fig.3, is rotating at $\omega = 4 \text{ rad/s}$ at the instant $\theta = 60^\circ$. If the coefficient of static friction at A is $\mu_s = 0.5$. Determine if the disk slips at the instant. $I = 0.51168 \text{ kg} \cdot \text{m}^2$ is moment of inertia of the semicircular about its mass center.

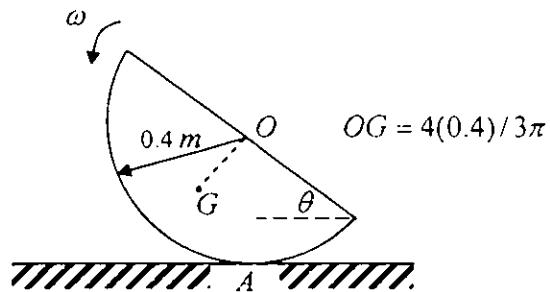


Figure 3

4. The ball has a mass of 8 kg and radius $r = 100 \text{ mm}$ and rolls without slipping on the horizontal surface at $v_G = 6 \text{ m/s}$, Fig.4. Determine the angular velocity of the ball and the normal force the ball exerts on the track when it reaches the position $\theta = 70^\circ$. Take $R = 500 \text{ mm}$. The moment of inertia of the ball about its mass center is $\frac{2}{5}mr^2$.



Figure 4

5. Find the natural frequency of the system shown in Fig 5.

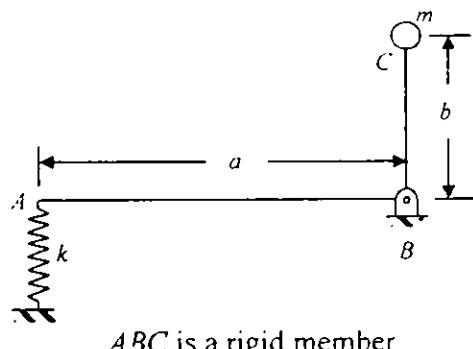


Figure 5

Rectilinear Motion

$$a = \frac{dv}{dt} = \frac{d^2x}{dt^2} = v \frac{dv}{dx} \quad x = x_0 + v_0 t + \frac{1}{2} a t^2 \quad v = v_0 + at \quad v^2 = v_0^2 + 2a(x - x_0)$$

Curvilinear Motion

$$\vec{v} = \frac{d\vec{r}}{dt} \quad \vec{r} = x\vec{i} + y\vec{j} + z\vec{k} \quad \vec{e}_r = \frac{d\vec{e}_t}{d\theta} \quad \vec{e}_t = \dot{\theta}\vec{e}_\theta \quad \vec{e}_\theta = -\dot{\theta}\vec{e}_t,$$

$$\vec{a} = \frac{d\vec{v}}{dt} \quad \vec{v} = \dot{x}\vec{i} + \dot{y}\vec{j} + \dot{z}\vec{k} \quad \vec{v} = v\vec{e}_t \quad \vec{v} = \dot{r}\vec{e}_r + r\dot{\theta}\vec{e}_\theta$$

$$\vec{a} = \ddot{x}\vec{i} + \ddot{y}\vec{j} + \ddot{z}\vec{k} \quad \vec{a} = \frac{dv}{dt}\vec{e}_t + \frac{v^2}{\rho}\vec{e}_n$$

$$\vec{a} = (\ddot{r} - r\dot{\theta}^2)\vec{e}_r + (r\ddot{\theta} + 2\dot{r}\dot{\theta})\vec{e}_\theta$$

$$\vec{r}_B = \vec{r}_A + \vec{r}_{B/A} \quad \vec{v}_B = \vec{v}_A + \vec{v}_{B/A} \quad \vec{a}_B = \vec{a}_A + \vec{a}_{B/A}$$

Kinetics of Particles

$$\sum \vec{F} = m\vec{a}$$

$$\sum F_x = ma_x \quad \sum F_t = ma_t \quad \sum F_r = ma_r$$

$$\sum F_y = ma_y \quad \sum F_n = ma_n \quad \sum F_\theta = ma_\theta$$

$$\sum F_z = ma_z \quad \sum F_i = ma_i$$

$$V_e = \frac{1}{2}k\alpha^2 \quad V_g = mgh \quad V_g = -\frac{mgR^2}{r} \quad T = \frac{1}{2}mv^2$$

$$T_1 + U_{1-2} = T_2 \quad T_1 + V_1 = T_2 + V_2 \quad \text{Power} = \vec{F} \cdot \vec{v}$$

$$\vec{L} = m\vec{v} \quad \sum \vec{F} = \vec{L} \quad \int_1^2 \sum \vec{F} dt = \vec{L}_2 - \vec{L}_1$$

$$\vec{H}_O = \vec{r} \times m\vec{v} \quad \sum \vec{M}_O = \vec{H}_O \quad \int_1^2 \sum \vec{M}_O dt = \vec{H}_{O_2} - \vec{H}_{O_1}$$

Systems of Particles

$$\sum \vec{F} = m\vec{a} \quad \vec{L} = \sum m\vec{v} = m\vec{v} \quad \sum \vec{F} = \vec{L} \quad \int_1^2 \sum \vec{F} dt = \vec{L}_2 - \vec{L}_1$$

$$\vec{H} = \sum \vec{r} \times m\vec{v} \quad \sum \vec{M}_O = \vec{H}_O \quad \sum \vec{M}_G = \vec{H}_G \quad \int_1^2 \sum \vec{M}_O dt = (\vec{H}_O)_2 - (\vec{H}_O)_1$$

Kinematics of Rigid Bodies

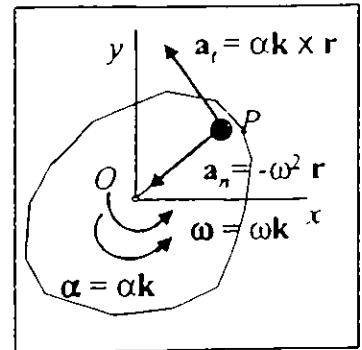
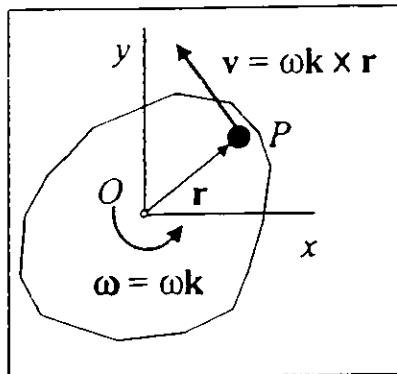
$$\omega = \frac{d\theta}{dt} = \dot{\theta}$$

$$a = \frac{d\omega}{dt} = \dot{\omega} \quad \text{or} \quad a = \frac{d^2\theta}{dt^2} = \ddot{\theta}$$

$$\omega = ad\theta \quad \text{or} \quad \dot{\theta} d\dot{\theta} = \ddot{\theta} d\theta$$

$$v = r\omega$$

$$a_n = r\omega^2 \quad a_t = ra$$



$$\vec{v}_A = \vec{v}_B + \vec{v}_{A/B}$$

$$v_{A/B} = r\omega$$

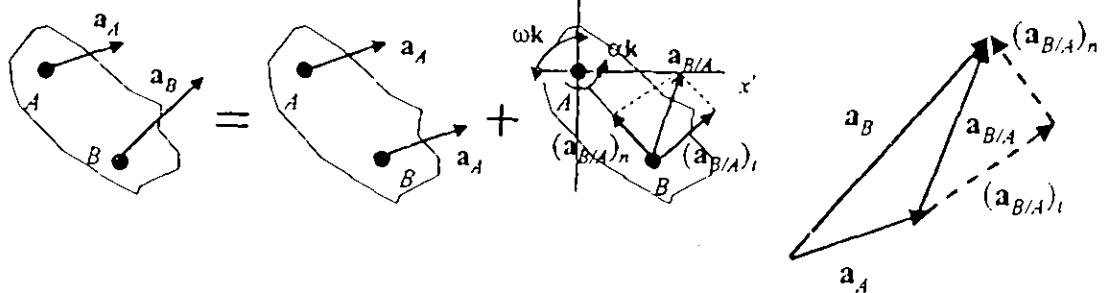
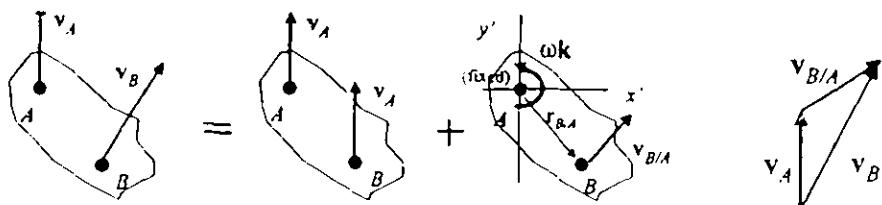
$$\vec{v}_{A/B} = \vec{\omega} \times \vec{r}$$

$$\vec{a}_A = \vec{a}_B + \vec{a}_{A/B}$$

$$\vec{a}_A = \vec{a}_B + (\vec{a}_{A/B})_n + (\vec{a}_{A/B})_t$$

$$(a_{A/B})_n = \frac{v_{A/B}^2}{r} = r\omega^2$$

$$(a_{A/B})_t = \dot{v}_{A/B} = r\alpha$$



Kinetics of Rigid Bodies

Equations of Motion

$$\sum F_x = m \ddot{a}_x \quad \sum F_y = m \ddot{a}_y \quad \sum M_G = I \ddot{a} \quad \sum M_o = I_o \alpha$$

Energy

$$T = \frac{1}{2} I_o \omega^2 \quad T = \frac{1}{2} m v_G^2 + \frac{1}{2} I_G \omega^2 \quad T_1 + \sum U_{1..2} = T_2$$

Impulse and Momentum

$$\bar{L} = m \vec{v} \quad \sum \bar{F} = \vec{\bar{L}} \quad \int_1^2 \bar{F} dt = \bar{L}_2 - \bar{L}_1$$

$$H_O = I_O \omega \quad \sum \bar{M}_O = \vec{H}_O \quad \int_1^2 \sum \bar{M}_O dt = I_O (\omega_2 - \omega_1)$$

$$H_G = \bar{I} \omega \quad \sum \bar{M}_G = \vec{H}_G \quad \int_1^2 \sum \bar{M}_G dt = (\bar{H}_G)_2 - (\bar{H}_G)_1$$

Free Vibration

$$m\ddot{x} + c\dot{x} + kx = 0 \quad \omega_n = \sqrt{\frac{k}{m}} \quad c_c = 2m\sqrt{\frac{k}{m}} = 2m\omega_n \quad \omega_d = \omega_n \sqrt{1 - \left(\frac{c}{c_c}\right)^2}$$

$$m\lambda^2 + c\lambda + k = 0 \quad \lambda_1 = -\frac{c}{2m} + \sqrt{\left(\frac{c}{2m}\right)^2 - \frac{k}{m}} \quad \lambda_2 = -\frac{c}{2m} - \sqrt{\left(\frac{c}{2m}\right)^2 - \frac{k}{m}}$$

$c > c_c$ Overdamped $x = A e^{\lambda_1 t} + B e^{\lambda_2 t}$ $c = c_c$ Critically damped $x = (A + Bt) e^{-\omega_n t}$

$c < c_c$ Underdamped $x = D [e^{-(c/2m)t} \sin(\omega_d t + \phi)]$

$$\text{log decrement} \quad \delta = \ln\left(\frac{x_1}{x_2}\right) = \frac{2\pi\left(\frac{c}{c_c}\right)}{\sqrt{1 - \left(\frac{c}{c_c}\right)^2}}$$

Forced Vibration

$$m\ddot{x} + c\dot{x} + kx = P_m \sin(\omega_f t) \quad x_p = X \sin(\omega_f t - \phi)$$

$$M = \frac{X}{P_m/k} = \frac{1}{\sqrt{[1 - (\omega_f/\omega_n)^2]^2 + [2(\frac{c}{c_c})(\omega_f/\omega_n)]^2}}$$

$$\phi = \tan^{-1} \left[\frac{2(\frac{c}{c_c})\omega_f/\omega_n}{1 - (\omega_f/\omega_n)^2} \right]$$