

# PHYS 121 — HW2 Solutions

October 31, 2025

## Q1. Spider-Man Swing (Tension Strength)

Bottom of arc (radius  $L$ ). Forces: tension  $T$  up along string, weight  $mg$  down. Centripetal:  $T - mg = mv^2/L$ . Hence

$$T = m \frac{v^2}{L} + mg.$$

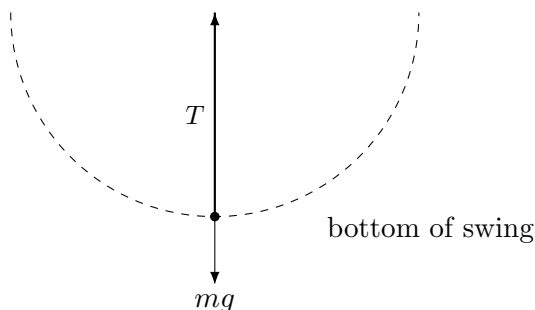
Assume traffic speed  $\approx 13$  m/s, so  $v \approx 26$  m/s; take  $m = 75$  kg,  $L \approx 75$  m (25 floors). Then

$$T \approx 75 \cdot \frac{26^2}{75} + 75 \cdot 9.8 \approx 1410 \text{ N}.$$

Minimum silk diameter from allowable stress  $\sigma$  (take dragline silk  $\sigma \approx 1.0 \times 10^9$  Pa): area  $A = T/\sigma$ ,  $d = 2\sqrt{A/\pi}$ :

$$A \approx 1.41 \times 10^{-6} \text{ m}^2, \quad \boxed{d_{\min} \approx 1.34 \text{ mm}}.$$

For equal safety, the better cord is the one with higher allowable stress (smaller required diameter). Dragline silk ( $\sim 1$  GPa) outperforms typical nylon rope ( $\sim 0.08 - 0.10$  GPa).



## Q2. Motorboat Terminal Speed

Given  $m = 190$  kg, thrust  $F_T = 40.0$  N, linear drag  $bv$  with  $b = 2$  N s/m. Newton's 2nd law gives

$$m \frac{dv}{dt} = F_T - bv.$$

(i) Rearrange to isolate  $v$  and  $t$ :  $\frac{dv}{dt} = \frac{F_T}{m} - \frac{b}{m}v$ .

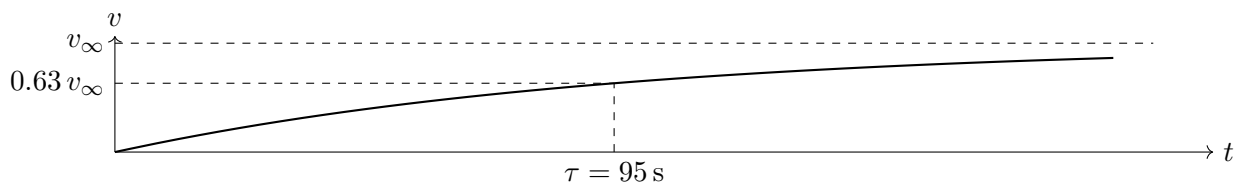
(ii) Define the terminal speed  $v_\infty = \frac{F_T}{b}$  so that  $\frac{dv}{dt} = \frac{b}{m}(v_\infty - v)$ .

(iii) Separate variables:  $\frac{dv}{v_\infty - v} = \frac{b}{m} dt$  and integrate from 0 to  $t$  (start from rest):

$$-\ln\left(1 - \frac{v}{v_\infty}\right) = \frac{b}{m} t.$$

(iv) Solve for  $v(t)$ :  $v(t) = v_\infty \left(1 - e^{-(b/m)t}\right)$ .

Numerics:  $v_\infty = \frac{40.0}{2} = \boxed{20 \text{ m/s}}$ , and  $\frac{b}{m} = \frac{2}{190} = \frac{1}{95}$ . The time when  $v = 0.63 v_\infty$  is  $t \approx \boxed{95 \text{ s}}$  (since  $e^{-1} = 0.37$ ). Checks: (1) As  $t \rightarrow \infty$ ,  $e^{-(b/m)t} \rightarrow 0$ , so  $v \rightarrow v_\infty$ . (2) At  $t \rightarrow 0$ ,  $\frac{dv}{dt} = \frac{b}{m} v_\infty = \frac{F_T}{m}$ , equal to  $F_{\text{net}}/m$  initially (drag = 0 at rest). Initial acceleration =  $\boxed{0.211 \text{ m/s}^2}$ .



### Q3. Box up an Incline (constant speed)

Incline  $30^\circ$ ,  $m = 45 \text{ kg}$ ,  $\mu_k = 0.35$ , horizontal  $F$ . Along the plane (up positive):

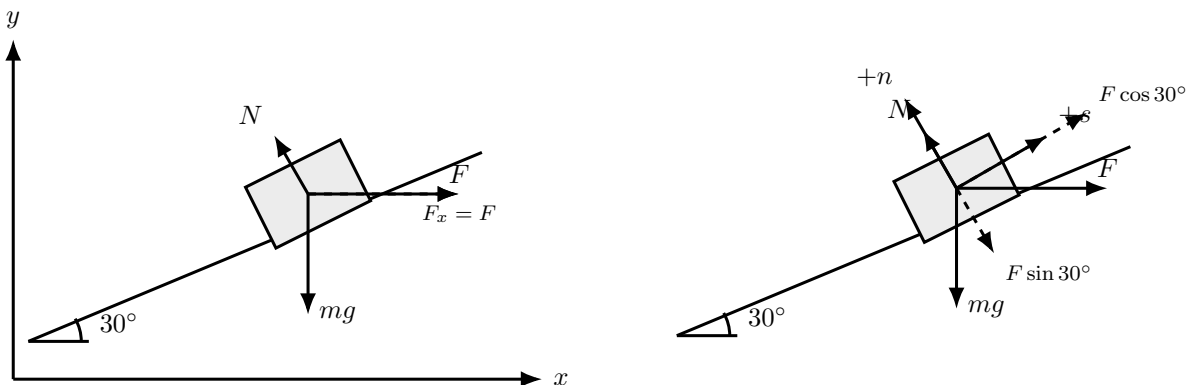
$$F \cos 30^\circ = mg \sin 30^\circ + \mu_k (mg \cos 30^\circ + F \sin 30^\circ).$$

Solve:  $F = \frac{mg(\sin 30^\circ + \mu_k \cos 30^\circ)}{\cos 30^\circ - \mu_k \sin 30^\circ} = \boxed{5.13 \times 10^2 \text{ N}}$ . Distance  $s = 8.0 \text{ m}$ . Work:

$$W_F = F s \cos 30^\circ = \boxed{3.55 \times 10^3 \text{ J}}, \quad N = mg \cos 30^\circ + F \sin 30^\circ \Rightarrow f_k = \mu_k N \approx 224 \text{ N},$$

$$W_f = -f_k s = \boxed{-1.79 \times 10^3 \text{ J}}, \quad W_g = -mg(s \sin 30^\circ) = \boxed{-1.76 \times 10^3 \text{ J}}.$$

*Signs:*  $W_F > 0$  because  $\vec{F}$  has a component along the displacement up the plane;  $W_f < 0$  since kinetic friction opposes motion;  $W_g < 0$  since gravity's component is down the plane while displacement is up the plane;  $W_N = 0$  because  $\vec{N} \perp$  displacement. *Check (Work-Energy):*  $W_{\text{net}} = \Delta K$ . Constant speed  $\Rightarrow \Delta K = 0 \Rightarrow W_{\text{net}} = 0$ , which our totals satisfy. Thus  $W_{\text{net}} \approx 0$  consistent with constant speed.



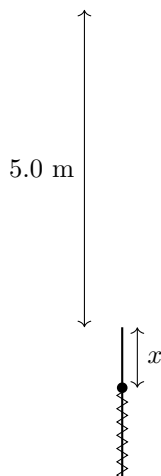
*Incline-aligned view is better for writing component equations; world-axes view is good for visualization.*

## Q4. Vertical Spring Gun

$k = 14 \text{ N/cm} = 1400 \text{ N/m}$ ,  $m = 15 \text{ g}$ , top is  $5.0 \text{ m}$  above the uncompressed end. Energy from compressed to top (include rise by  $x$  while leaving):

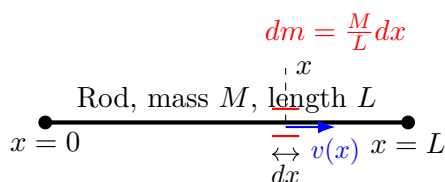
$$\frac{1}{2}kx^2 = mg(5.0 + x).$$

Quadratic  $700x^2 - 0.147x - 0.735 = 0$  gives physical root  $x \approx 3.25 \text{ cm}$ .



## Q5. Kinetic Energy of a Massive Spring

**Rod model:** Consider a rod of length  $L$  and total mass  $M$ . At position  $x$  (measured from one end,  $0 \leq x \leq L$ ), take a slice of length  $dx$ . The mass of this slice is  $dm = (M/L)dx$ , where  $M/L$  is the linear mass density. If the rod has a linear speed profile  $v(x) = (x/L)v$  (where  $v$  is the speed at the far end  $x = L$ ), then each slice moves with speed  $v(x)$ .



**Kinetic energy element:** The kinetic energy of the slice at position  $x$  is

$$dK = \frac{1}{2}dm v(x)^2 = \frac{1}{2} \frac{M}{L} \left( \frac{x}{L}v \right)^2 dx = \frac{1}{2} \frac{Mv^2}{L^3} x^2 dx.$$

**Integration:** The total kinetic energy is obtained by integrating  $dK$  from  $x = 0$  to  $x = L$ :

$$K = \int_0^L dK = \frac{Mv^2}{2L^3} \int_0^L x^2 dx = \frac{Mv^2}{2L^3} \cdot \frac{L^3}{3} = \boxed{\frac{1}{6}Mv^2}.$$

**Comparison with point-mass result:** For a point mass of mass  $M$  moving at speed  $v$ , the kinetic energy would be  $K_{\text{point}} = \frac{1}{2}Mv^2$ . Our result is  $\frac{1}{6}Mv^2$ , which is exactly *one-third* of the point-mass result.

**Explanation:** The difference arises because the rod has a *distribution of speeds*. In our model, only the far end ( $x = L$ ) moves at speed  $v$ ; parts closer to  $x = 0$  move slower (e.g., the center at  $x = L/2$  moves at  $v/2$ ). Since kinetic energy depends on  $v^2$ , the slower-moving parts contribute less energy than if the entire mass were moving at speed  $v$ . The factor of  $1/3$  reflects the quadratic averaging over the linear speed profile:  $\langle v^2 \rangle = \frac{1}{L} \int_0^L \left(\frac{x}{L}v\right)^2 dx = \frac{v^2}{3}$ .

## Q6. Power, Drag, and Fuel

At  $v = 15$  m/s, power to wheels  $P_{15} = 20$  hp  $= 1.492 \times 10^4$  W. (i) Constant drag  $F_d$ :

$$F_d = \frac{P_{15}}{v} = \boxed{9.95 \times 10^2 \text{ N}}, \quad P_{30} = F_d (30 \text{ m/s}) = \boxed{2.98 \times 10^4 \text{ W}} \quad (\approx 40 \text{ hp}).$$

Energy for 10 km:  $W = F_d d = \boxed{9.95 \times 10^6 \text{ J}}$  to wheels; fuel energy (25% efficiency):  $\boxed{3.98 \times 10^7 \text{ J}}$  at either speed.

(ii) Linear drag  $F_d = kv$  matched at 15 m/s:  $k = F_d/v = \boxed{66.3 \text{ N s/m}}$ . Then

$$P(v) = kv^2, \quad P_{30} = k(30)^2 = \boxed{5.97 \times 10^4 \text{ W}} \quad (\approx 80 \text{ hp}).$$

Energy for 10 km:  $W = kvd$  gives  $W_{15} = \boxed{9.95 \times 10^6 \text{ J}}$ ,  $W_{30} = \boxed{1.99 \times 10^7 \text{ J}}$ ; fuel energies are  $4 \times$  these.

**Reflection:** In (i) doubling speed doubles power and leaves energy per distance unchanged; in (ii) doubling speed quadruples power and doubles energy per distance. Real driving matches the trend that higher speed needs much more power and fuel per mile. Aerodynamic drag is approximately quadratic,  $F \propto v^2$ , so realistically  $P \propto v^3$  and energy per distance  $\propto v^2$ , which increases even faster with speed than our linear-drag model predicts.

## Q7. Peg and Complete Loop

Release from horizontal, length  $a$ . After catching the peg, the small-circle radius is  $a - h$ .

- Energy (release  $\rightarrow$  bottom of big circle, drop  $a$ ):

$$mga = \frac{1}{2}mv_b^2 \quad \Rightarrow \quad v_b^2 = 2ga, \quad v_b = \sqrt{2ga}.$$

- Energy (bottom  $\rightarrow$  top of small circle, rise  $2(a - h)$ ):

$$\frac{1}{2}mv_b^2 = \frac{1}{2}mv_t^2 + mg \cdot 2(a - h) \quad \Rightarrow \quad v_t^2 = v_b^2 - 4g(a - h) = 2ga - 4g(a - h), \quad v_t = \sqrt{2ga - 4g(a - h)}.$$

Non-slack at the top requires a strictly positive tension. At the top, taking inward as positive, the radial balance is

$$T + mg = m \frac{v_t^2}{a - h} \quad \Rightarrow \quad T = m \frac{v_t^2}{a - h} - mg.$$

Requiring  $T > 0$  gives

$$v_t^2 > g(a - h).$$

Using  $v_t^2 = 2ga - 4g(a - h)$  gives  $2a - 4(a - h) > (a - h)$ , i.e.,  $-2a + 4h > a - h$ . Hence  $5h > 3a$ , so  $\boxed{h > \frac{3}{5}a}$ .

## Q8. Bungee Jump Estimate

Mass 72 kg, free fall 15 m before stretch; spring  $k = 50 \text{ N/m}$ . Energy from platform to lowest point:

$$mg(15 + x) = \frac{1}{2}kx^2.$$

Solve  $25x^2 - 705.6x - 10584 = 0$  for  $x > 0$ :  $x \approx 39.1 \text{ m}$ . Total drop  $15 + x \approx 54.1 \text{ m}$ .



## Q9. Bullet + Block (inelastic)

$m_b = 0.220 \text{ kg}$  at  $v_i = 400 \text{ m/s}$  embeds into  $M = 1.30 \text{ kg}$  on a frictionless table. Momentum conservation:

$$v_f = \frac{m_b v_i}{m_b + M} = 57.9 \text{ m/s (east)}.$$

Impulses:  $J_b = m_b(v_f - v_i) = -75.3 \text{ N s}$  (on bullet, west),  $J_M = Mv_f = 75.3 \text{ N s}$  (on block, east).

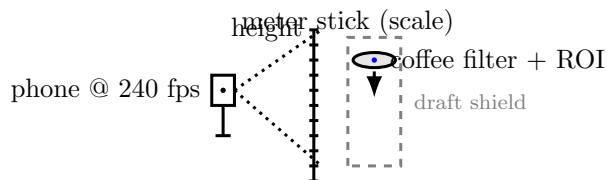
Over 3 ms:  $\bar{F} \approx 2.51 \times 10^4 \text{ N}$ .



Vectors drawn to scale:  $p_b = p_f = 88 \text{ N s}$  east. With  $m_b = 0.220 \text{ kg}$ ,  $v_i = 400 \text{ m/s}$ ,  
 $(m_b + M)v_f = 88 \text{ N s} \Rightarrow v_f = 57.9 \text{ m/s}$ .

*Note: Q10 omitted per course announcement.*

## Bonus. Terminal-speed experiment (0–2 pts)



- **Setup:** Indoors, drop a coffee filter beside a vertical meter stick; phone on tripod  $\geq 240 \text{ fps}$ , optical axis perpendicular, use a clear bin as a draft shield and a colored ROI dot on the filter.
- **Data:** Track  $y(t)$  of the ROI every 2–4 frames across 3–5 drops; repeat with 1–4 stacked filters to vary mass and confirm  $v_t$  scaling.

- **Model/fit:** Quadratic drag gives  $v(t) = v_t \tanh(gt/v_t)$  and the position model  $y(t) = y_0 + \frac{v_t^2}{g} \operatorname{Incosh}(gt/v_t)$ ; fit  $y(t)$  directly (less noisy) to extract  $v_t$ . Linear-drag cross-check:  $v(t) = v_t(1 - e^{-t/\tau})$ .
- **Uncertainty:** Systematic (parallax, scale skew, timing) and random (drafts, tilt). Mitigate via long camera distance, careful alignment, averaging trials, and verifying that late-time  $v(t)$  plateaus.

## Assistance notes:

LLMs were used as translators and concept explainers. No direct solutions were provided.