

# Project 1 Report - ME155C/ECE147C

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## **Abstract**

In this project, we design a controller for a two mass - one spring, cart system. The goal is to optimize the performance of the second cart's response to a step input to the reference through control of the first cart's behavior, with respect to several key control metrics. We look at optimizing for a very short settling time along with minimizing the amount of overshoot. The 'undershoot' is treated as an acceptable behavior since the cart will pass through this region, so we give ourselves the difficulty of treating the overshoot as a highly undesirable set. We achieve our goals with a settling time under three seconds along with less than one percent overshoot.

# 1 Introduction

We begin our control journey with the search of an optimal controller. The first idea here is LQR, but it is clear that we have noise in our system and uncertainty in our data, so we consider going to LQG. We try to decrease our uncertainty by doing a sinusoidal sweep of 20 frequencies.

## 2 System Identification

### 2.1 Process to be controlled

The process we are controlling is a two cart system connected by a spring. The system is driven by a motor that applies a force  $F$  to the first cart with mass  $m_1$  [kg], and the second cart with mass  $m_2$  [kg] is connected to the first cart via a spring with spring constant  $k$  [N/m]. In this system,  $x_1$  [m] is the position of the first cart, and  $x_2$  [m] is the position of the second cart. The control input is the voltage  $u := V$  [Volt] applied to the motor, and the measured output is the position  $y := x_2$  [m] of the second cart.

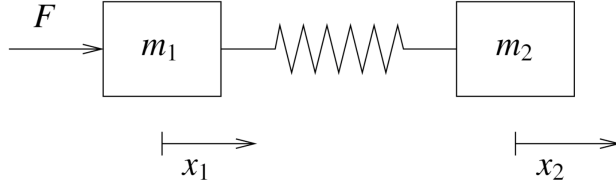


Figure 1: Two cart system

### 2.2 Non-Parametric Identification

The non-parametric identification method used was sine-wave testing in conjunction with the correlation method. This strategy consists of applying sinusoidal inputs at distinct frequencies to calculate the magnitude and phase of the frequency response at that frequency from the output of the system.

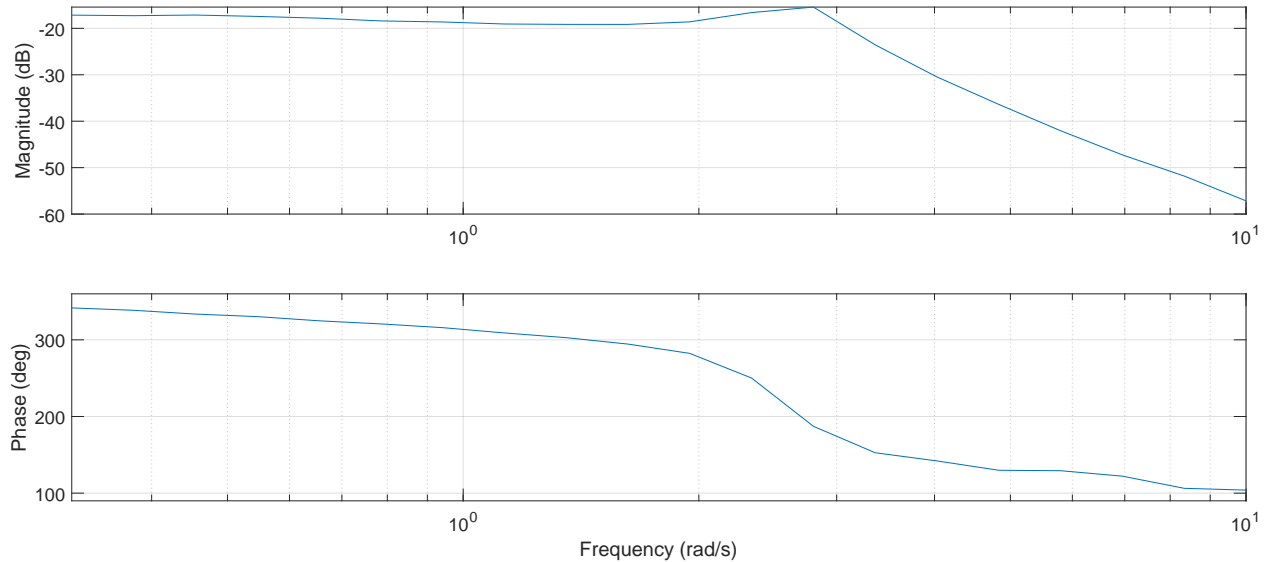


Figure 2: Bode plot of the identified system using non-parametric identification

### 2.3 Parametric Identification

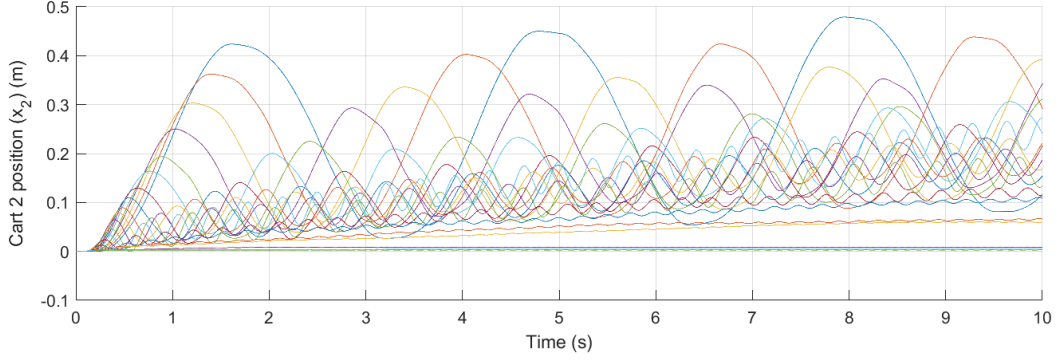


Figure 3: Output signals of all experiments for parametric identification

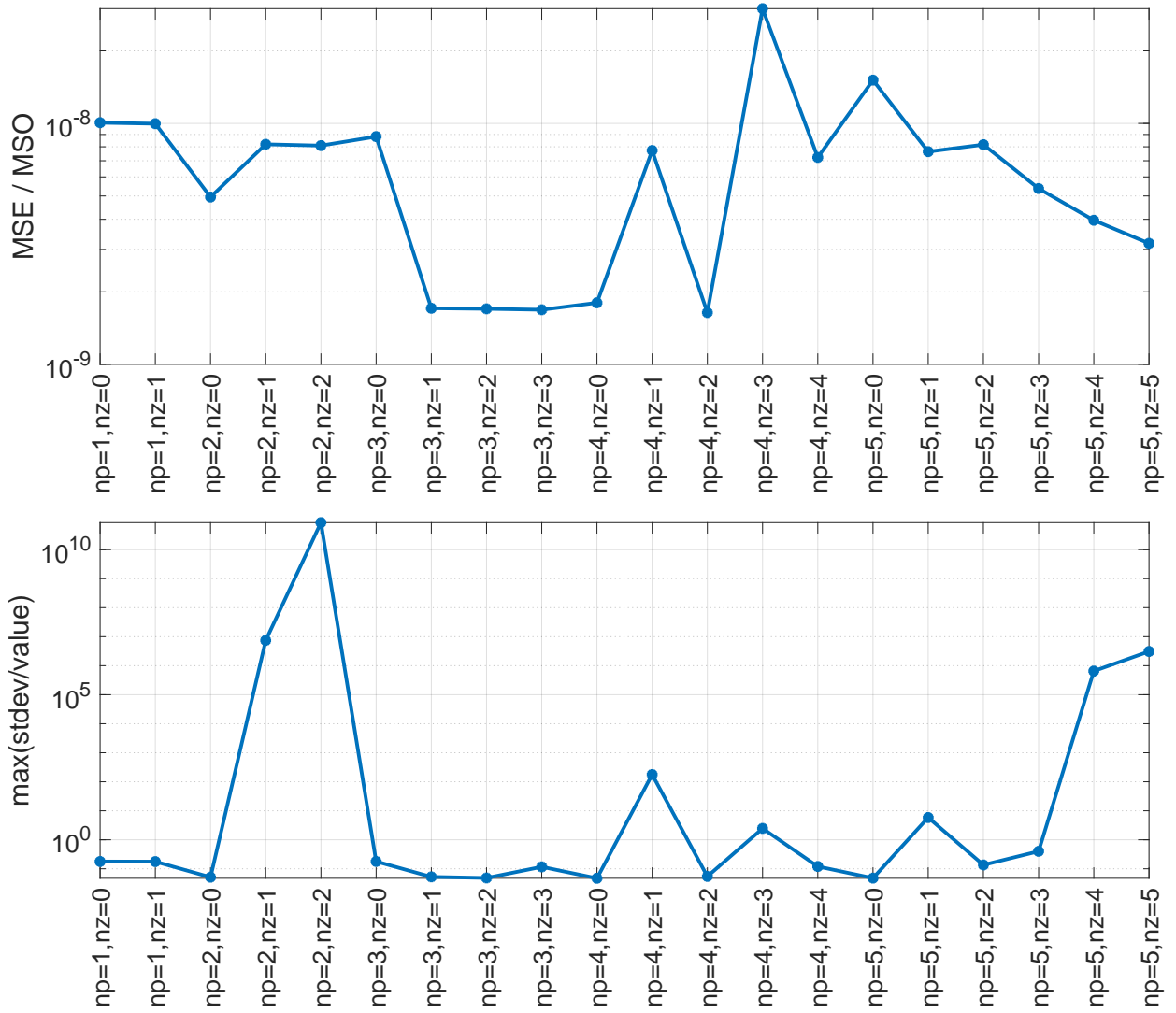


Figure 4: Normalized MSE and worst parameter standard deviation for different model orders

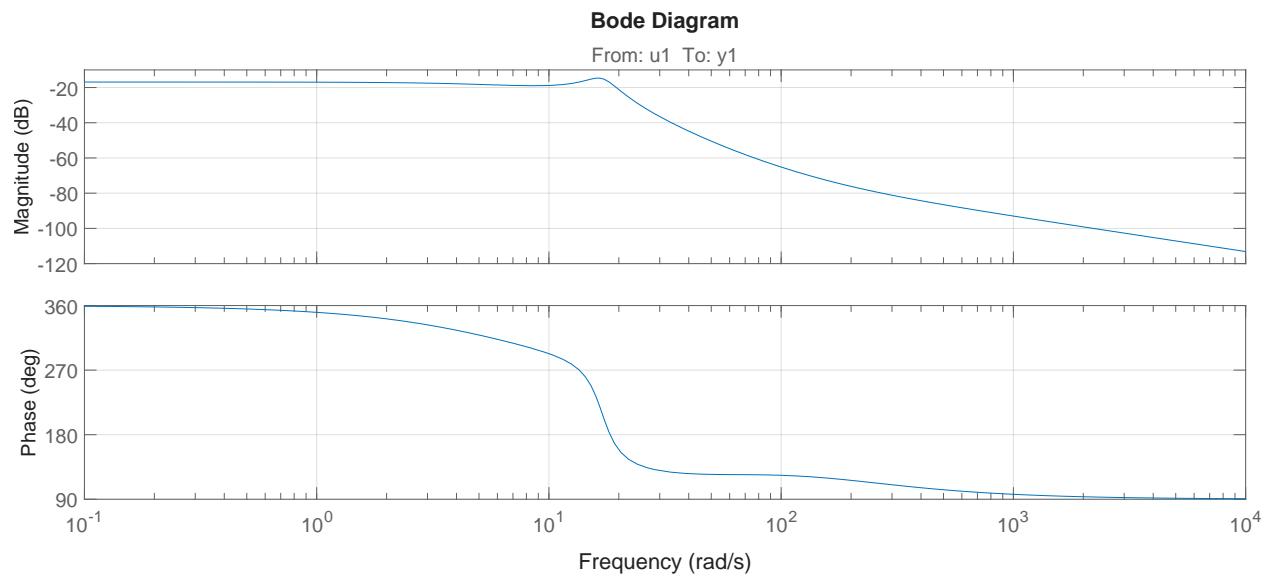


Figure 5: Bode plot of the identified system using parametric identification

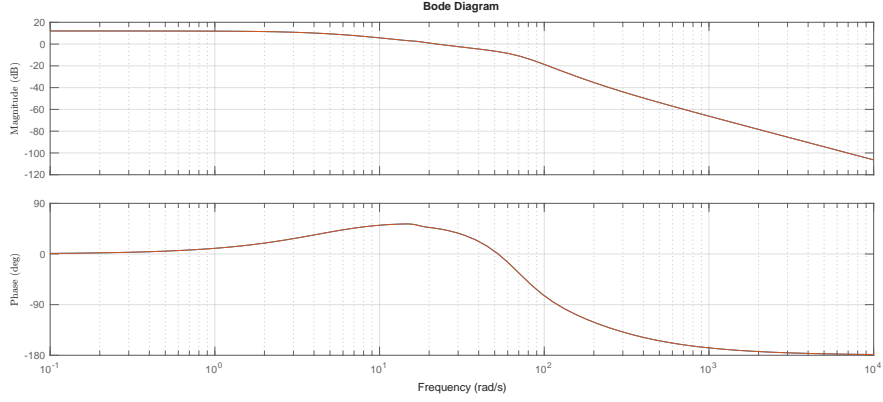


Figure 6: Closed-loop frequency response (simulated)

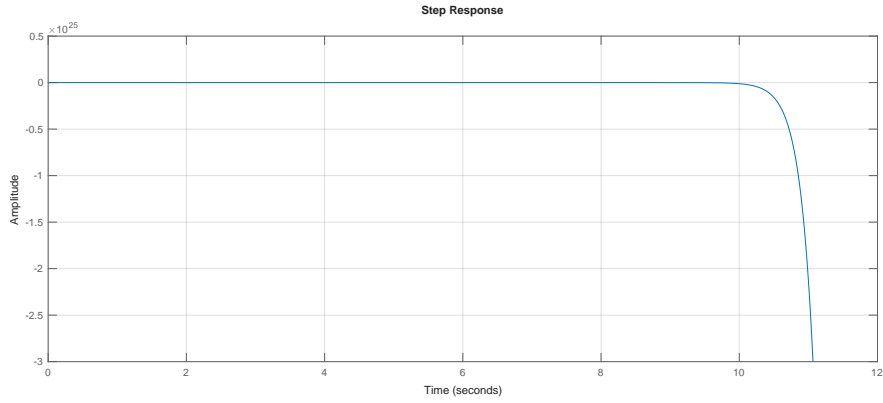


Figure 7: Closed-loop step response (simulated)

### 3 Controller Design

#### 3.1 Design Methodology

Next, we were tasked to design a controller for the system. We shot for an overshoot of less than 15% and a settling time of less than 2 seconds. To achieve this, we chose to use an LQR/LQG design. Using our identified model, we used Matlab to calculate the optimal LQR gains and Kalman filter.

However, even with the optimal linear controller, the unmodelled dead zone of the motor still caused there to be a large steady state error. To fix this, we added an integrator to the controller, which successfully eliminated the steady state error. The final controller is an LQR/LQG controller in parallel with an integrator, which led to a closed loop system with an adequate performance.

#### 3.2 Simulation Results

We simulated the closed-loop system in Simulink using the identified model. The frequency response can be seen in Figure 8. The simulated step response is unstable, as seen in Figure 7.

## 4 Closed-loop Testing

For the frequency response identification of the closed loop system, we ran a sinusoidal sweep across frequencies:  $\omega \in 2\pi[10^{-0.5}, 10]$ . The theoretical and experimental bode diagrams seem to disagree in phase and magnitude. This can be due to an inaccurate estimation of the order of the transfer function but ultimately we end up with a real closed loop that performs well so it is not too important.

### 4.1 Step Response Experiments

Parameter	Value
Overshoot	0.3%
Rise Time	1.81 sec
Settling Time	2.23 sec
Max Control Input	5 V

Table 1: Step Response Data

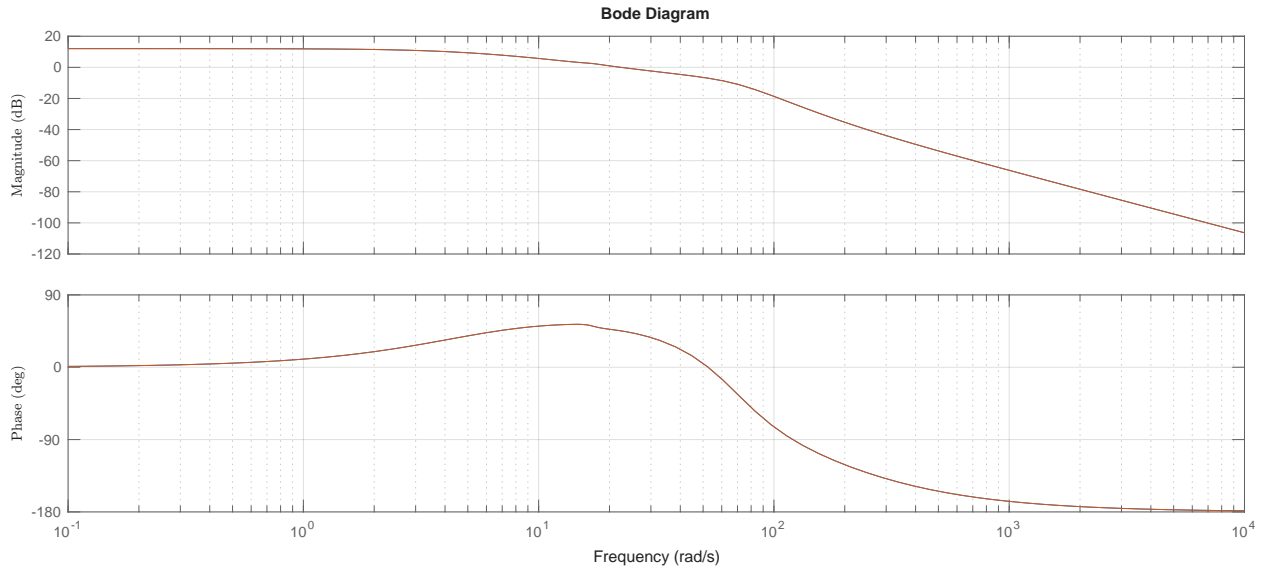


Figure 8: Bode plot of complementary sensitivity function of simulated controller

### 4.2 Closed-loop Frequency Response

### 4.3 Closed-loop Frequency Response

## 5 Conclusions and Future Work

We are happy with the performance of our controller and would love to improve it in the following ways. First, we ran a small sweep of frequencies but if we had more time, it may make sense to test many more frequencies with a finer mesh. Additionally, we could test a wider range to get better data about the noise and for higher frequency data. The sinusoidal sweep tests are simple, but it would also be interesting to try a test using a white noise input. Next, for the controller

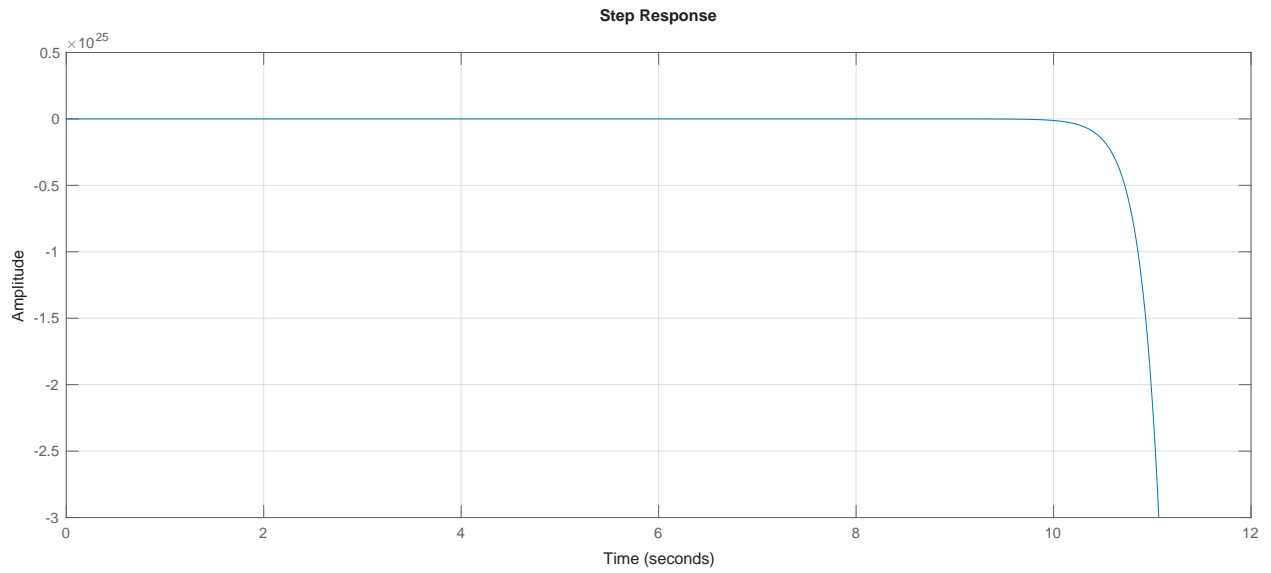


Figure 9: Simulated closed-loop step response

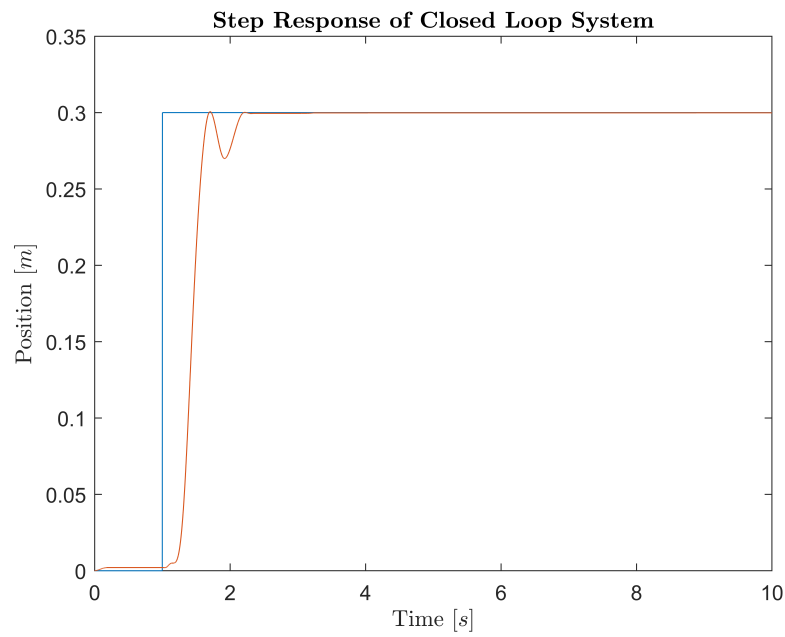


Figure 10: Closed-loop step response

performance, we see that around resonance there is a large tracking error. This is a place which could use more work, but part of the issues could come from optimizing for that ideal step response behavior.

## References



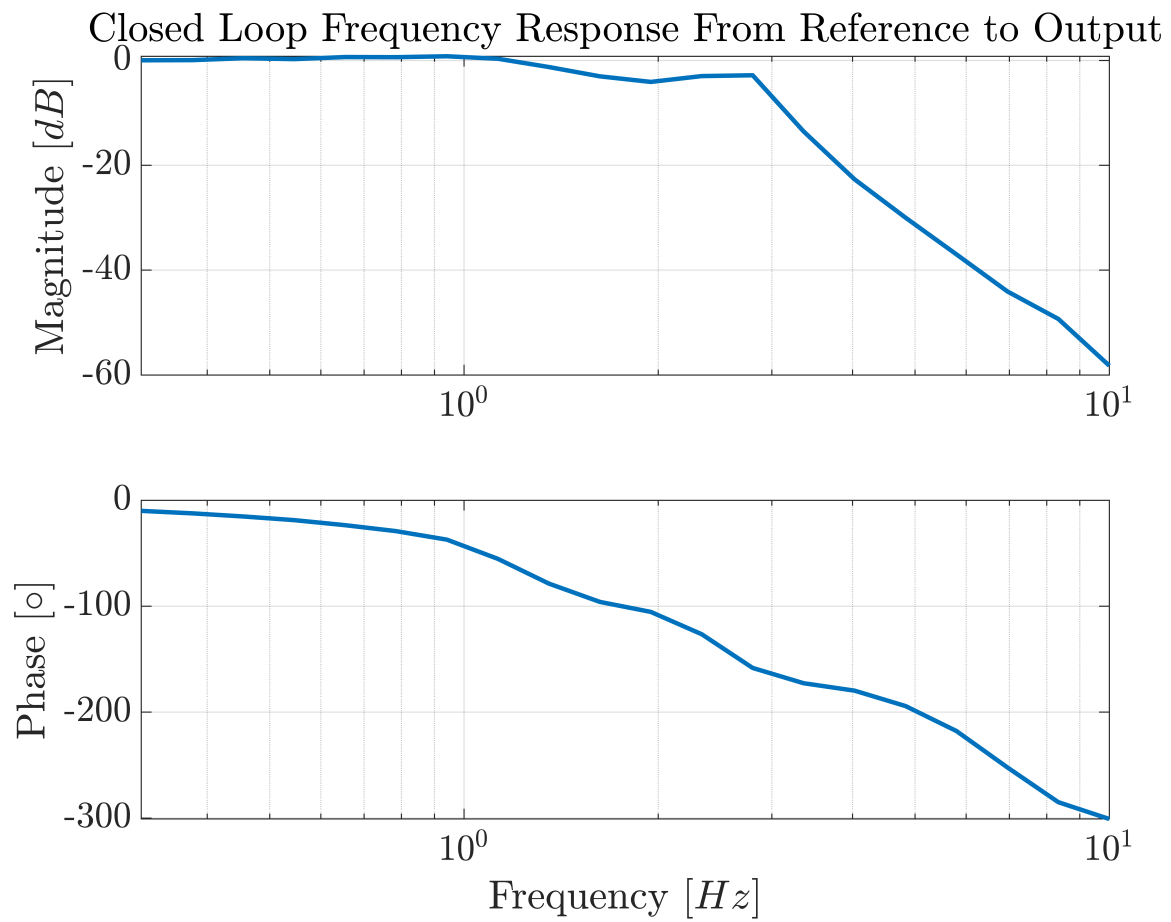


Figure 11: Closed-loop frequency response of the transfer function from reference to output

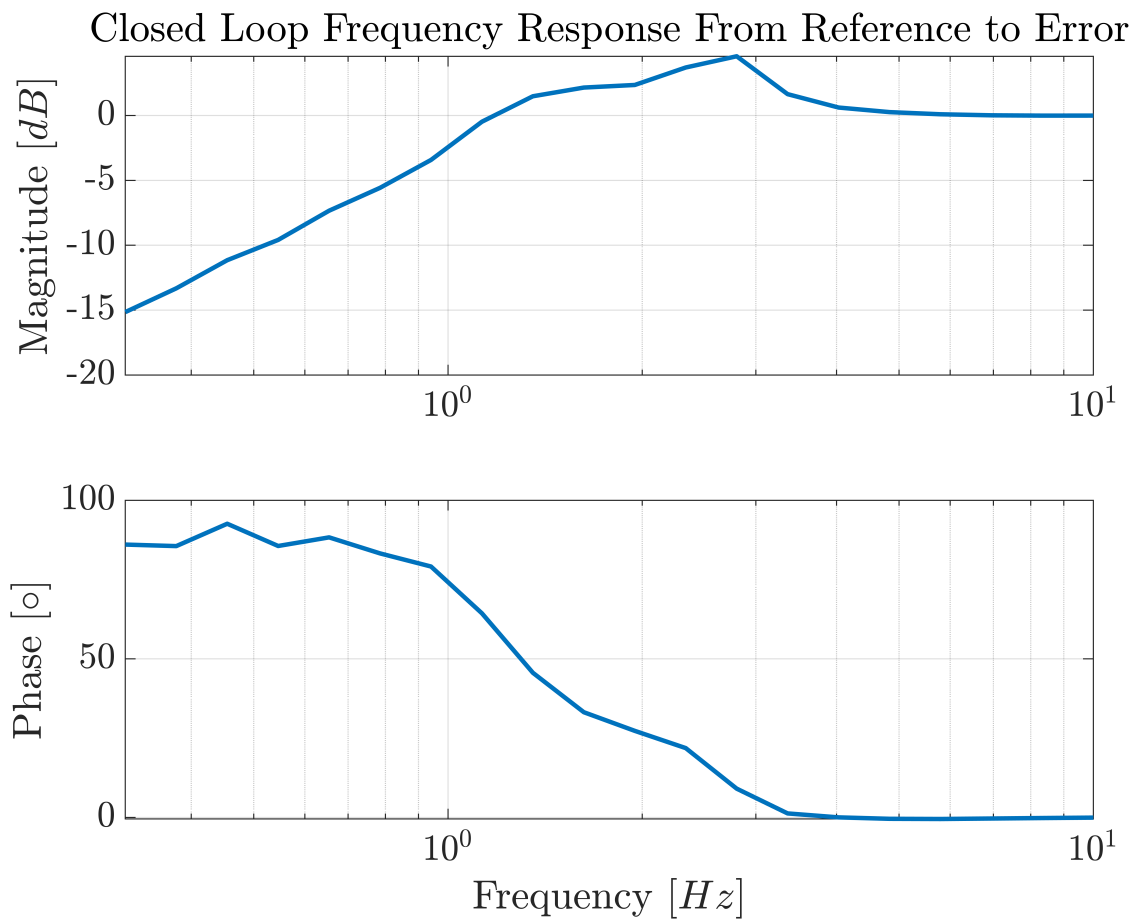


Figure 12: Closed-loop frequency response of the transfer function from reference to error