

Thermo

Recitation #6

(03/14/23)

→ Session 1 : key concepts & Exam 2 pointers

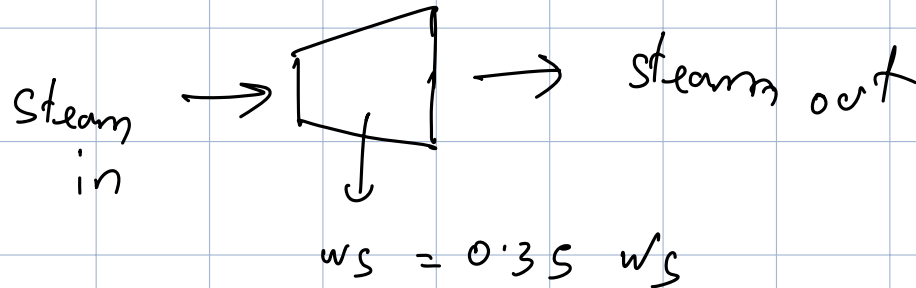
Session 2 : Problems: 1 entropy prob., 1 turbine prob.

Session 1

Key concepts

① Reversibility

H.W #4.3



(a) Adiabatic & reversible $\equiv \Delta S = 0$
isentropic

i.e. $\eta = 100\%$

Path variables $\rightarrow Q \text{ \& } W$

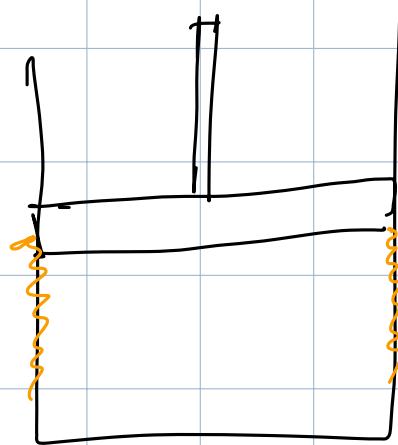
$$\Delta S = \frac{Q_{\text{rev}}}{T} \quad \vdots \quad \frac{dS}{dt} = \frac{\dot{Q}_{\text{rev}}}{T}$$

if irreversible.

$$\Delta S = \frac{Q_{\text{irrev}}}{T} + \underbrace{\dot{S}_{\text{gen}}}$$

Physical interpretation

irreversibility



friction on wall makes the process irrev.

when adiabatic : $\dot{Q} = 0$

for reversible process $\Delta S = \frac{\dot{Q}_{rev}}{T} = 0$

$\therefore \Delta S = 0$ of adiabatic & reversible process

(b) $\eta = 35\%$

$$\Delta U = \cancel{Q}^0 + W_{ec}^0$$

$$\Delta U = 0$$

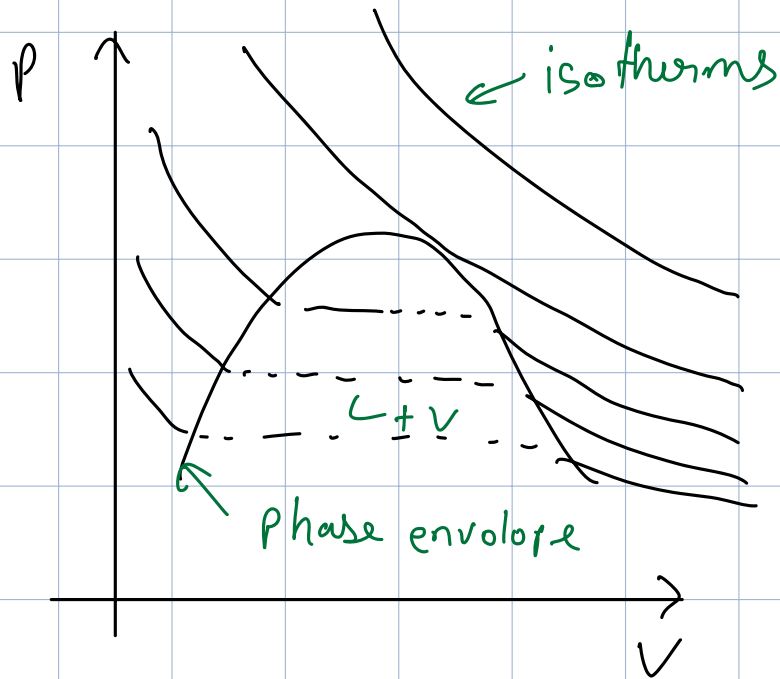
$$\therefore \Delta H' = \Delta \int U^0 + W_S'$$

$$\therefore \Delta H' = W_S'$$

$$\Delta H \cdot \eta = W_S \cdot \eta$$

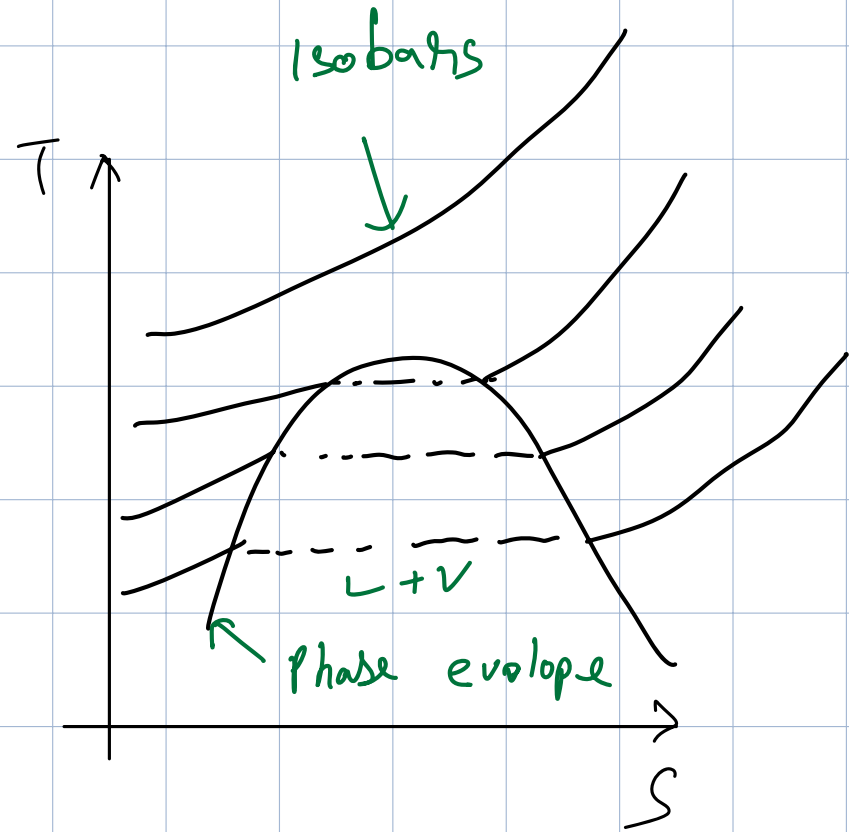
2

T-S diagram



$$V \propto \frac{1}{P} \quad (\text{Boyle's law})$$

$$\Rightarrow P_1 V_1 = P_2 V_2 \quad \leftarrow \text{governing eqns} \rightarrow \Rightarrow$$



$$S \propto -\frac{1}{T}$$

$$S = \frac{\dot{Q}_{rev.}}{T}$$

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Reversible vs. irreversible outlet.

following cases.

Reversible Outlet

superheated

wet steam

wet steam

using quality q for calculating superheated states

Actual Outlet

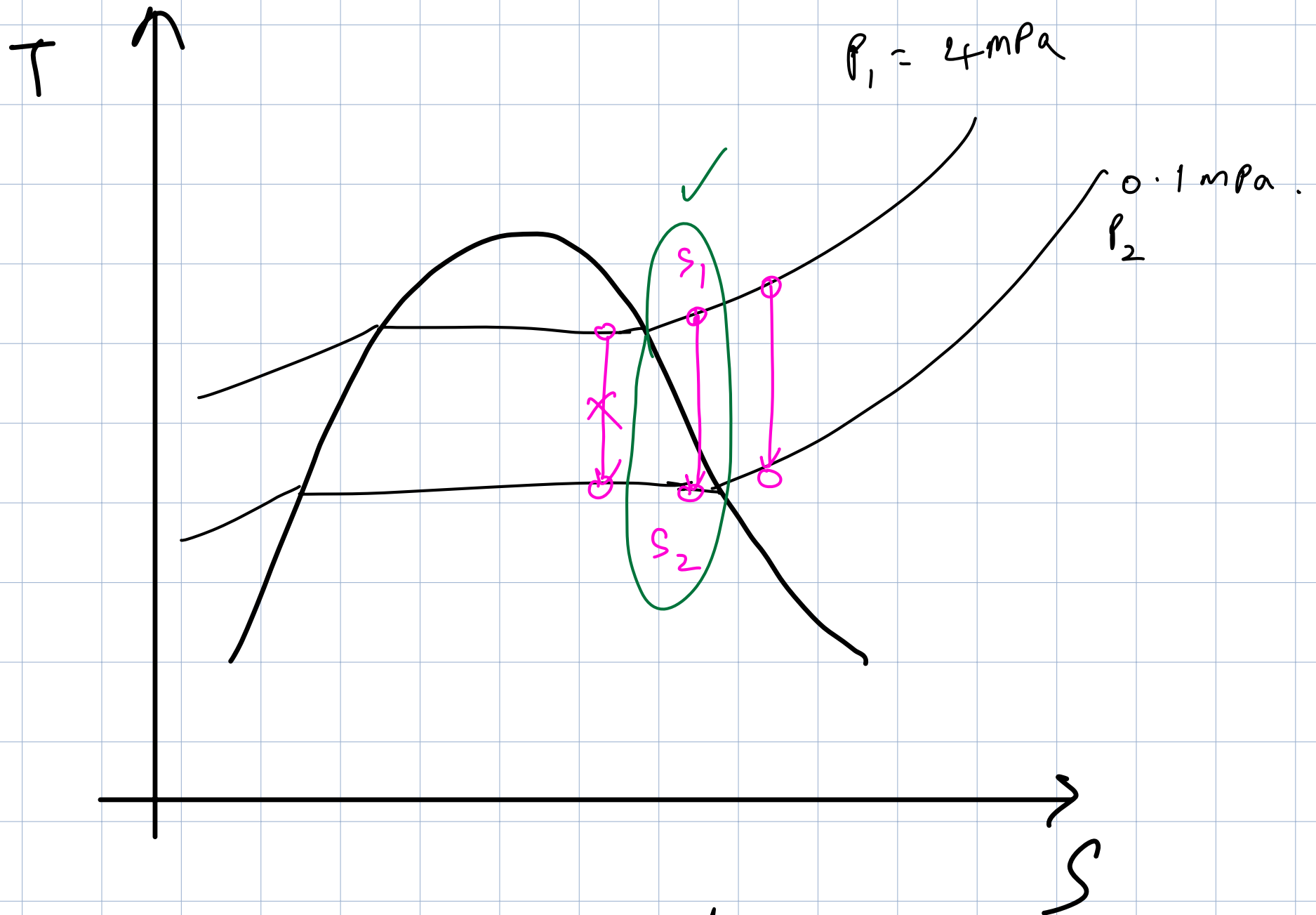
superheated

superheated

wet steam

when we talk about turbines: Adiabatic

if also reversible $\Rightarrow \Delta S = 0$



$T_1 = 500$ if Adiabatic & rev.
find steam quality.

$$s_1 = 7.09 \text{ kJ/kgK}$$

$$\therefore \Delta s = 0, \quad s_2 = s_1 = 7.09$$

$$\textcircled{a} \text{ } p_1: \quad T^{\text{sat}} = 99.6^\circ\text{C}$$

$$\text{A } s_f = 7.3589$$

look at sat pressure table

$$s_L = 1.3028$$

$$s_V = 7.3589$$

we see,

$$s_L < s_1 = s_2 < s_V$$

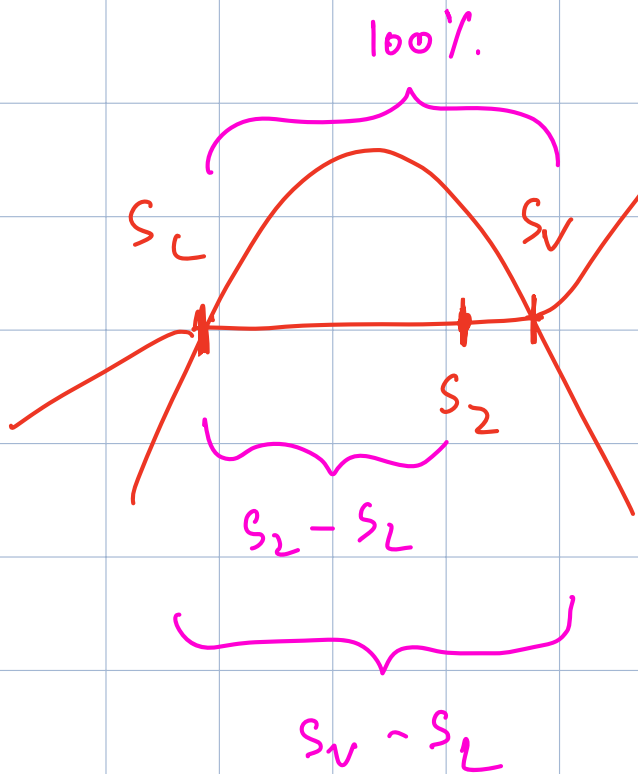
i. wet steam.

we know,

$$q = \frac{s_2 - s_L}{s_v - s_L} = \frac{7.09 - 1.3028}{7.3589 - 1.3028}$$

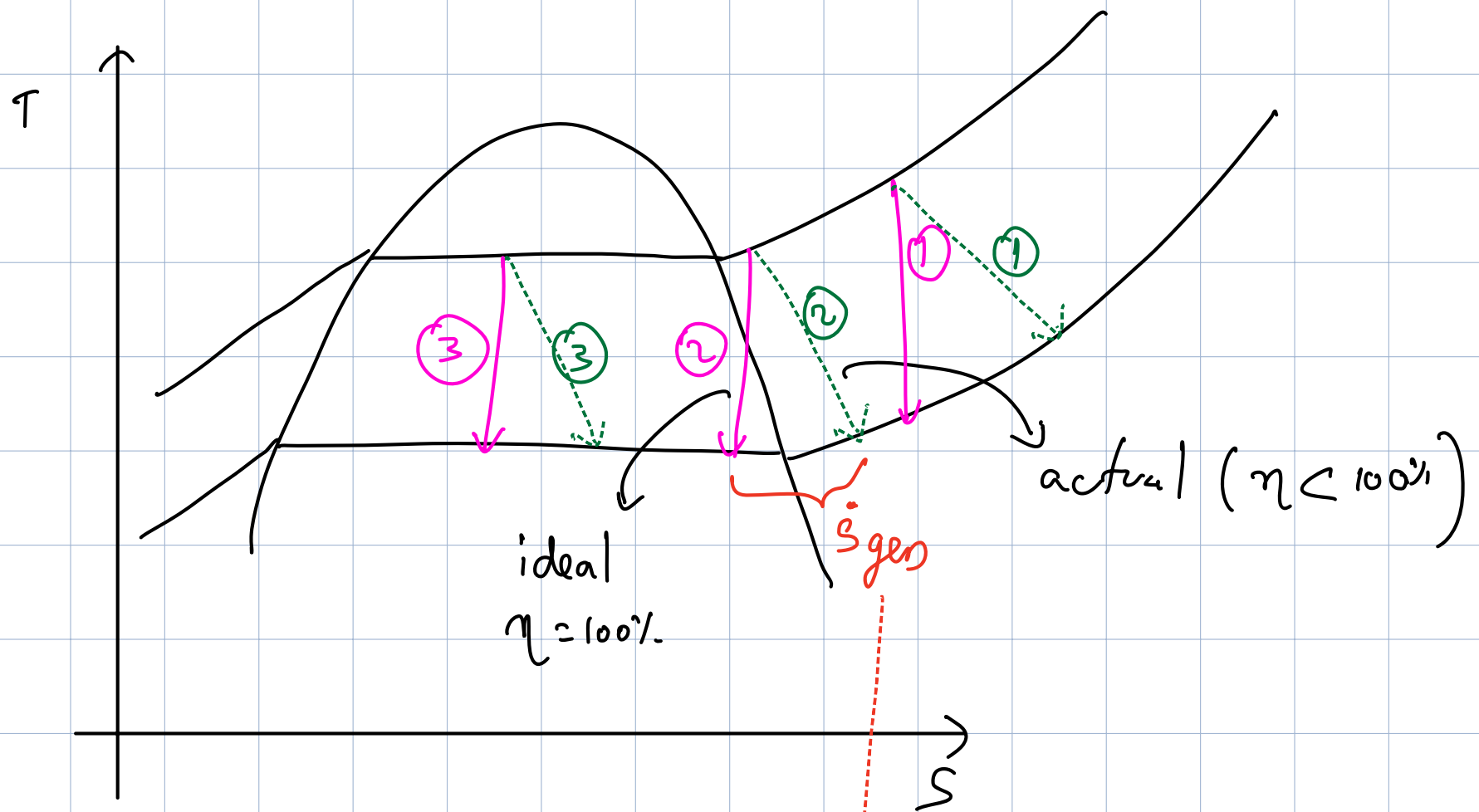
$$= 0.956$$

$$q = \underline{\underline{95.6\%}}$$



just ratio
of lengths.

Now if irrev. process



$$\Delta \dot{S} = \frac{Q_{irrev}}{T} + \dot{S}_{gen}.$$

Following cases.

Reversible Outlet

- ① superheated
- ② wet steam
- ③ wet steam

using quality x for calculating superheated states

Actual Outlet

- ① superheated
- ② superheated
- ③ wet steam

↑ from Exam 2 learning objective.

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Exam 2 pointers.

→ learning objective.

→ Exam pattern

<u>Q1.</u>	60 pts	OR	60 pts	11 - 11:45
<u>Q2</u>	30 pts		20 pts	11:45 - 12
<u>Q3.</u>	10 pts.		20 pts.	12 - 12:15

Focus on: entropy calc, irreversible/rev.
processes, T-S diagram.

→ Focus on understanding problems in H.Ws & Recitation.

→ Draw diagram & graph, Assumptions, governing equation \Rightarrow All these carry points.

→ Calculators, download material on your device. (you cannot use internet)

Session 2 : Problems

Q1

Reservoir

Gas

$$T_{\text{res}} = 30^{\circ}\text{C} = 323.15\text{K}$$

for gas

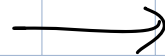
①

$$T_1 = 15^{\circ}\text{C}$$

$$P_1 = 0.1013\text{MPa}$$

$$V_1 = 60\text{m}^3$$

heated.



$$T_2 = 30^{\circ}\text{C}$$

$$P_2 = P_1$$

Assume : ideal gas

$$\text{no. of moles} = n = \frac{pV}{RT} = \frac{0.1013 \times 10^6 \times 60}{8.314 \times 288.15}$$

$$\underline{n = 2537.1 \text{ mol}}$$

$$(\Delta S_{\text{univ}})_p = (\Delta S_{\text{gas}})_p + (\Delta S_{\text{surrounding}})_p$$

$$\begin{aligned} (\Delta S_{\text{gas}})_p &= \frac{Q}{T} = \frac{n c_p dT}{T} = n \int_{T_1}^{T_2} \frac{c_p}{T} dT = n c_p \ln \left(\frac{T_2}{T_1} \right) \\ &= 2537.1 \times \frac{7R}{2} \ln \left(\frac{303.15}{288.15} \right) \end{aligned}$$

$$(\Delta S_{\text{gas}})_p = 3746.4 \text{ J/K}$$

$$\begin{aligned}
 (\Delta S_{\text{surr.}})_p &= \frac{Q_{\text{res}}}{T_{\text{res}}} = \frac{-Q_{\text{gas}}}{T_{\text{res}}} = \frac{-n C_p \Delta T}{T_{\text{res}}} \\
 &= \frac{-2537.1 \times \frac{7R}{2} (15)}{323.15}
 \end{aligned}$$

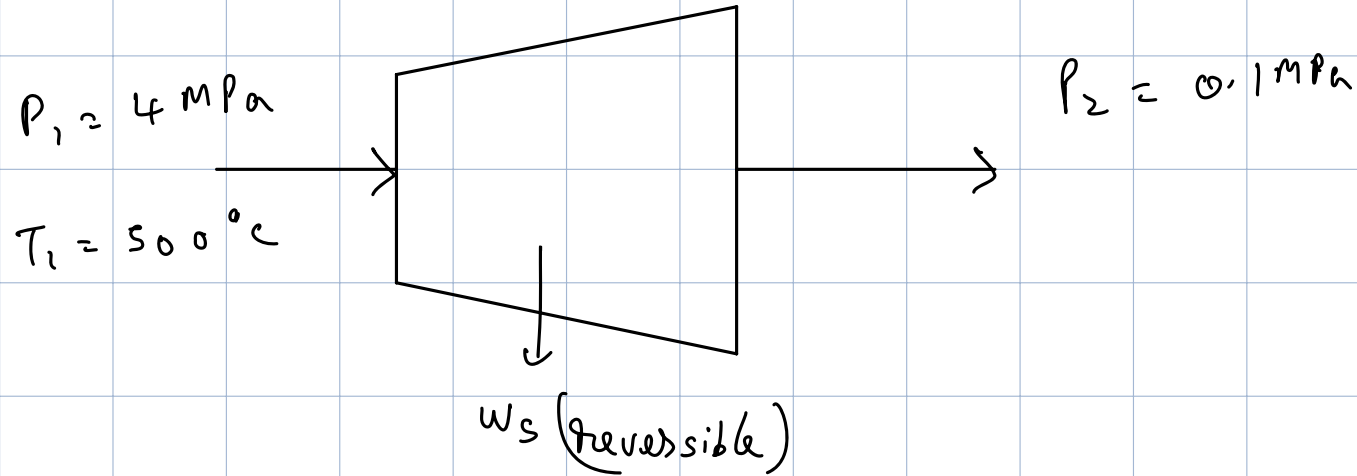
$$(\Delta S_{\text{surr}})_p = -3426.9 \text{ J/K}$$

$$\therefore (\Delta S_{\text{univ}})_p = 3746.4 + (-3426.9)$$

$$(\Delta S_{\text{univ}})_p = 319.5 \text{ J/K}$$

* Irreversible due to T gradient.

Q2



Assume : Steady - state
Adiabatic

ideal turbine $\rightarrow \underbrace{\dot{Q} = 0 \text{ \& \# 4 Reversible}}_{\Delta S = 0 \text{ \& \# isentropic}}$

(a) Mass balance : $\dot{m}_1 = -\dot{m}_2 = \dot{m}$

$$① \rho^{sat} = 0.1 \text{ MPa}$$

$$\left. \begin{array}{l} s^L = 1.3028 \\ s^V = 7.3589 \end{array} \right\} \text{kJ/kgK}$$

$$\left. \begin{array}{l} H^L = 417.50 \\ H^V = 2674.95 \end{array} \right\} \text{kJ/kg}$$

$$\therefore s^L < s_2' < s^V$$

interpolation of H @ ②

	S	H
L	x_1 1.3028	417.50 y_1
②	x 7.0922	2575.54 y
V	x_2 7.3589	2674.95 y_2

$$y = \frac{y_2 - y_1}{x_2 - x_1} (x - x_1) + y_1$$

$$= \frac{2674.95 - 417.50}{7.3589 - 1.3028} (7.0922 - 1.3028) + 417.50$$

$$+ 2674.95$$

$$= 2575.54$$

$$q = \frac{s_2' - s^L}{s^v - s^L} = \frac{7.0922 - 1.3028}{7.3589 - 1.3028} = 0.956$$

$$q = 95.6\%$$

$$H_2' = H^L + q \underbrace{(H^v - H^L)}_{\Delta H_{\text{vap}}}$$

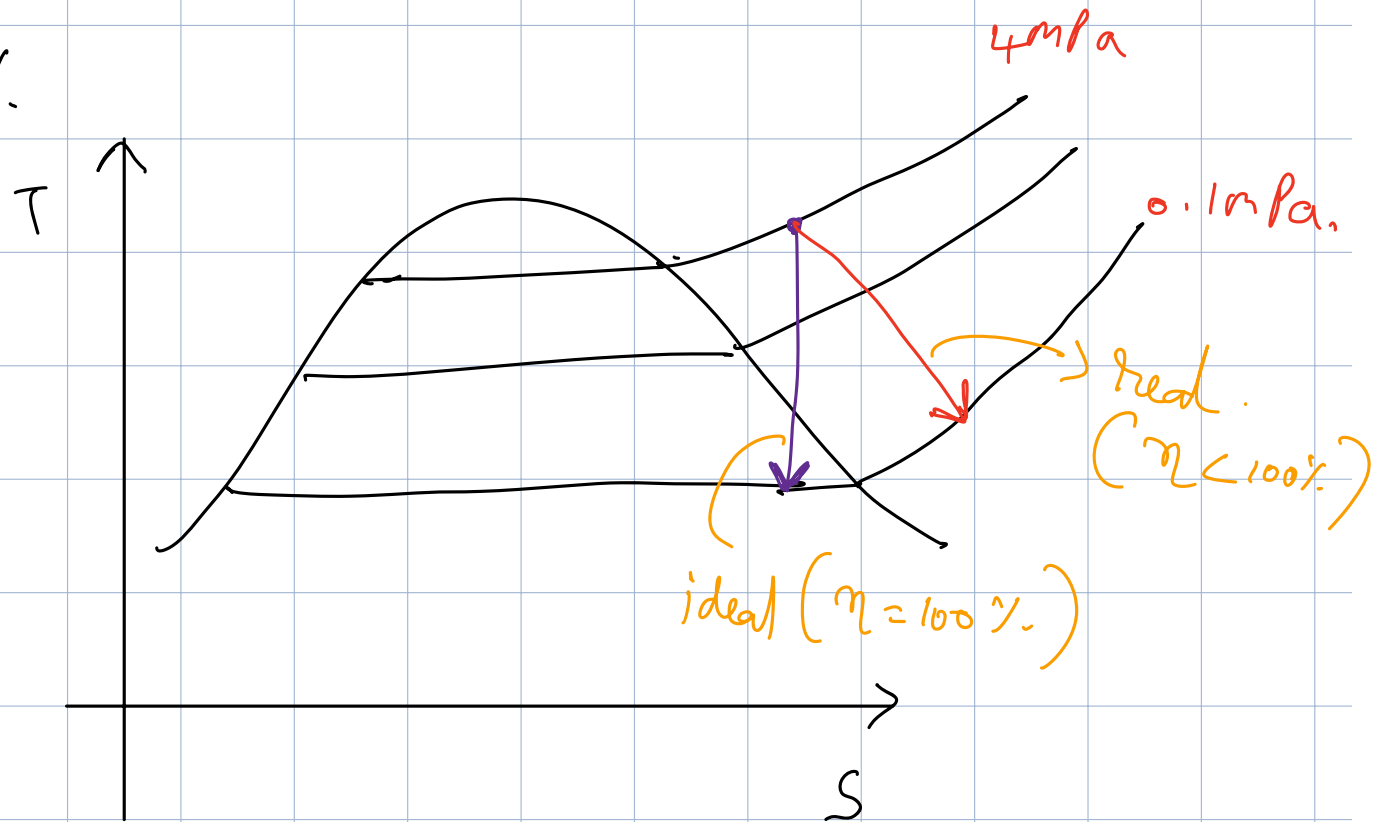
$$= 417.50 + 0.956 (2674.95 - 417.50)$$

$$H_2' = 2575.54 \text{ kJ/kg}$$

$$\Delta H' = w_{s'} = H_2' - H_1 = 2575.54 - 3446.0$$

$$\Delta H' = -870.46 \text{ kJ/kg}$$

b) if $\eta = 80\%$



$$\begin{aligned}
 w_s &= \Delta H' = \Delta H \cdot \eta = -870.46 \times 0.8 \\
 &= -696.37 \text{ kJ/kg}
 \end{aligned}$$