

The Finite Volume Method for Transport Equations

Chapter 1

MXB324 Computational Fluid Dynamics
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Weeks 1–2
July 2018



Outline

- 1 Introduction to FVM
 - 2 FVM in one dimension
 - 3 Diffusion equation
 - 4 Advection-diffusion equation
 - 5 Treatment of Nonlinearity
 - 6 Introduction to FVM in two dimensions
- Transport eq's*

Introduction

$$\frac{\partial \underline{\varphi}}{\partial t} + \nabla \cdot (\underline{\varphi} \underline{v}) = S$$

- The **Finite Volume Method** (FVM) is a method designed particularly for solving transport equations.
- In the field of computational fluid dynamics, especially in modern CFD software packages, it is much more widely used than the **finite difference method**. **F_LU_EN_T**.
- One important reason for its popularity is that it conserves, even at the discrete level. Thus, even though the numerical solution is only approximate, it nevertheless obeys the laws of conservation of mass, momentum and energy.
- Other desirable features of the method are its applicability on unstructured meshes and its natural handling of boundary conditions.



Meshes

- The basic geometric structure that underlies the finite volume method is called the **mesh**.
- A mesh consists of a set of **nodes**, and a set of connections between the nodes to form **elements**.
- Elements can be of any shape, but commonly used shapes are triangles and quadrilaterals in two dimensions, and tetrahedra and hexahedra in three dimensions.
- An **unstructured** mesh is one for which there is no regular grid underlying the mesh.
- These are difficult to construct (fortunately there is a wealth of software, both free and commercial, that can build these meshes), but very versatile—able to take the shape of complex domains and be refined in regions of interest.



Examples of Meshes

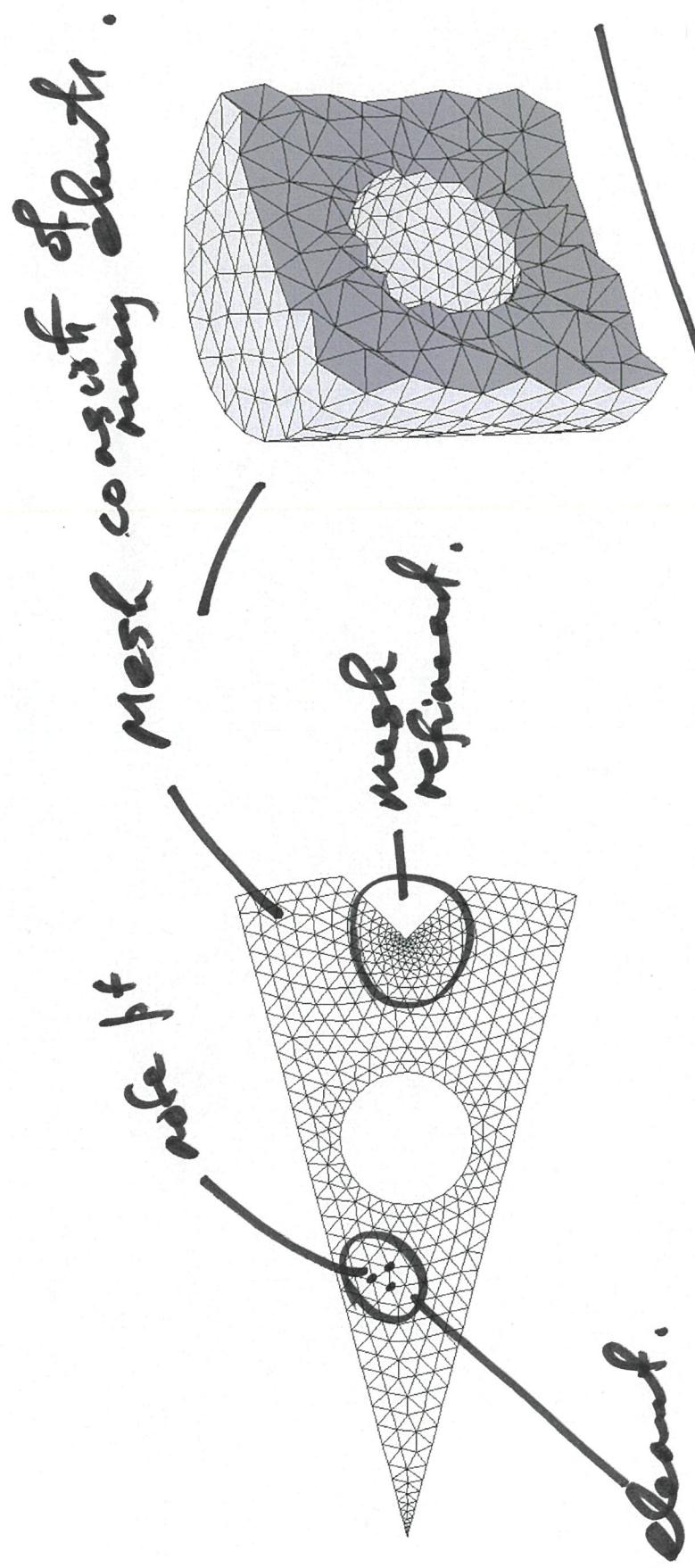


Figure: Unstructured meshes: (a) triangular; (b) tetrahedral.



Control Volume Construction

- The finite volume method uses the mesh to construct control volumes over which conservation is enforced.
 - ▶ Although technically these structures are only “volumes” when working in three dimensions, it is standard terminology even when working in one or two dimensions.
- There are a number of ways in which these control volumes can be constructed.
- In a cell-centred approach, the elements themselves form the control volumes, and a new set of nodes is constructed at the element centroids.
- In a vertex-centred approach, control volumes are constructed around the existing nodes, forming a **dual mesh** over the top of the existing mesh.



Illustration of Control Volume Construction in 3D

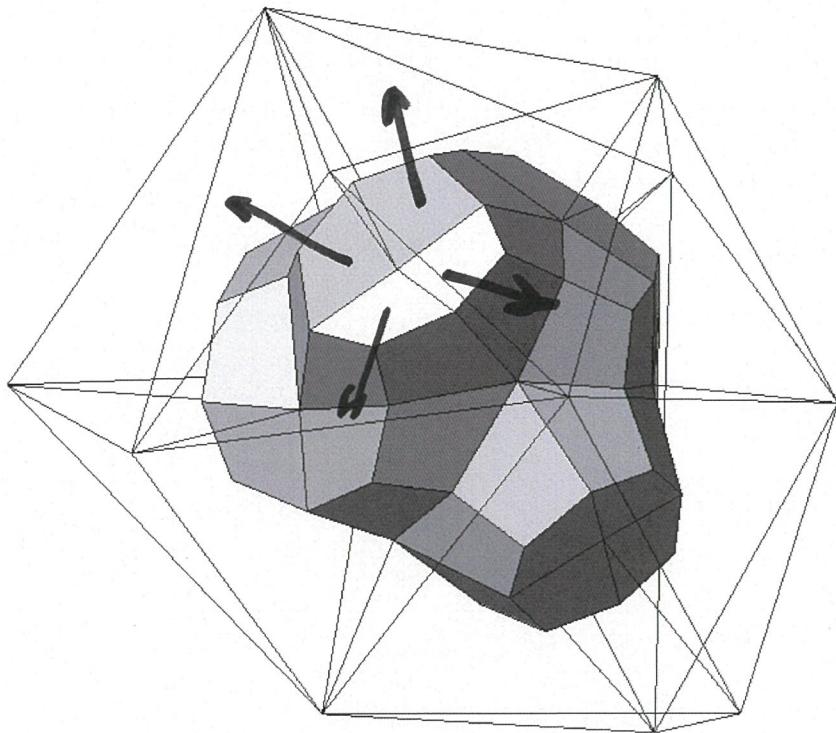


Figure: Vertex-centred control volume in three dimensions

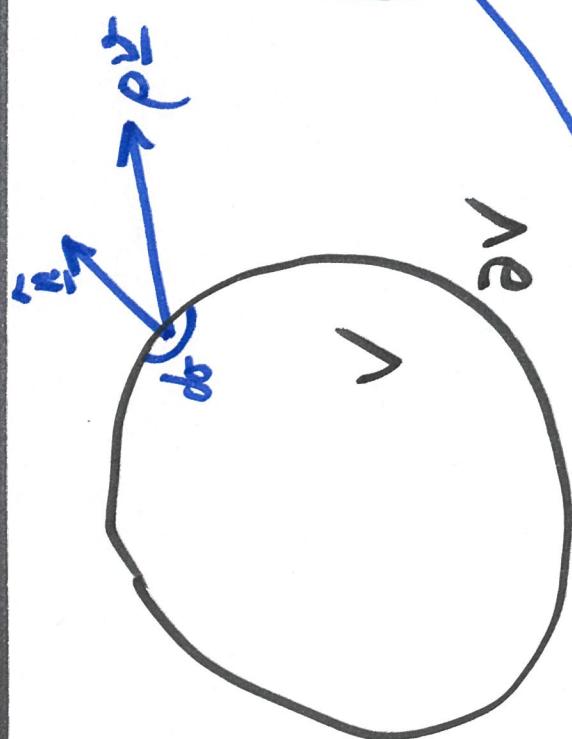
Control Volume Construction (continued)

Vertex - centres

Whichever approach is used to construct control volumes, the finite volume method works in the same way:

- 1 the PDE is integrated over each control volume,
- 2 numerical approximations are introduced,
- 3 the resulting system of equations is solved to yield a solution at each node.

Conservation of Mass :



Rate of change of Mass
in ∂V
= the mass leaving
into/out of
+ kinetic sources/sinks
of mass.

$$\frac{\partial}{\partial t} \int_V \rho(x, t) dV = - \oint_{\partial V} \rho(x, t) \frac{dS}{ds}$$

through
boundaries

R.E.V.
negative
discretizing
volume

$$\text{Divergence Theorem} \quad \int_V \left(\frac{\partial \rho}{\partial t} + \nabla \cdot (\rho \vec{v}) - S \right) dV = 0$$

$$+ \int_S \frac{\rho \vec{v} \cdot \hat{n}}{ds} dS = \int_S \frac{\rho \vec{v} \cdot \hat{n}}{ds} dS$$

Since \vec{v} is ambient; if must be
that

Conservation of Mass:

$$\frac{\partial \rho}{\partial t} + \nabla \cdot (\rho \mathbf{v}) = 0$$

$$\bar{\psi} = \frac{1}{\Delta V_i} \iint \psi dV \leftarrow \text{Average of fraction over } V_i$$
$$\bar{S} = \frac{1}{\Delta V_i} \iint S dV \leftarrow \text{Average of source over } V_i$$

General Transport Equation

We consider the general transport equation

$$\underbrace{\frac{\partial(\rho\varphi)}{\partial t} + \nabla \cdot (\rho\varphi \mathbf{v})}_{\text{advective term}} = \nabla \cdot (\Gamma \nabla \varphi) + \underbrace{S_\varphi}_{\substack{\text{Diffusive term} \\ \text{Source/Sink}}} \quad (1)$$

Or, as it is sometimes written for compactness,

$$\underbrace{\frac{\partial \psi}{\partial t}}_{\text{Unsteady term}} + \underbrace{\nabla \cdot \mathbf{J}}_{\text{Flux term}} = \underbrace{S_\varphi}_{\text{Source term}} \quad (2)$$

where $\psi = \rho\varphi$, and $\mathbf{J} = \rho\varphi \mathbf{v} - \Gamma \nabla \varphi$ is the flux vector, representing both advective and diffusive fluxes.

Note: capital \mathbf{J} is used, to avoid confusion with the coordinate vector \mathbf{j} .



Control Volume Formulation

If we integrate (2) over the i th control volume V_i , we obtain

$$\iiint_{V_i} \frac{\partial \psi}{\partial t} dV + \iiint_{V_i} \nabla \cdot \mathbf{J} dV = \iiint_{V_i} S_\varphi dV.$$

Rearranging, and applying the divergence theorem gives

$$\frac{d}{dt} \iiint_{V_i} \psi dV = - \iint_{\partial V_i} \hat{\mathbf{n}} \cdot \hat{\mathbf{J}} d\sigma + \iiint_{V_i} S_\varphi dV.$$

$\Delta V_i \cdot \bar{\nabla} \psi_i$

Control Volume Formulation

Defining the control volume averaged values of ψ and S_φ over V_i as $\bar{\psi}_i$ and $\bar{S}_{\varphi i}$ respectively, we obtain

$$\frac{d\bar{\psi}_i}{dt} \Delta V_i = - \underbrace{\iint_{\partial V_i} \mathbf{J} \cdot \hat{\mathbf{n}} d\sigma}_{\cancel{\text{or}}} + \overline{S_{\varphi i}} \Delta V_i$$

Or

$$\boxed{\frac{d\bar{\psi}_i}{dt} = -\frac{1}{\Delta V_i} \iint_{\partial V_i} \mathbf{J} \cdot \hat{\mathbf{n}} d\sigma + \overline{S_{\varphi i}}} \quad (3)$$

At this point, the first step is complete: (3) is an *exact reformulation* of (2), in control volume form.

Control Volume Formulation

To proceed we must introduce numerical approximations.

First, we replace the control volume averaged values $\bar{\psi}_i$ and \bar{S}_{φ_i} with their values at the control volume node, ψ_i and S_{φ_i} respectively, to obtain the following approximate form of (3):

$$\frac{d\psi_i}{dt} = \frac{-1}{\Delta V_i} \iint_{\partial V_i} \mathbf{J} \cdot \hat{\mathbf{n}} d\sigma + S_{\varphi_i}.$$

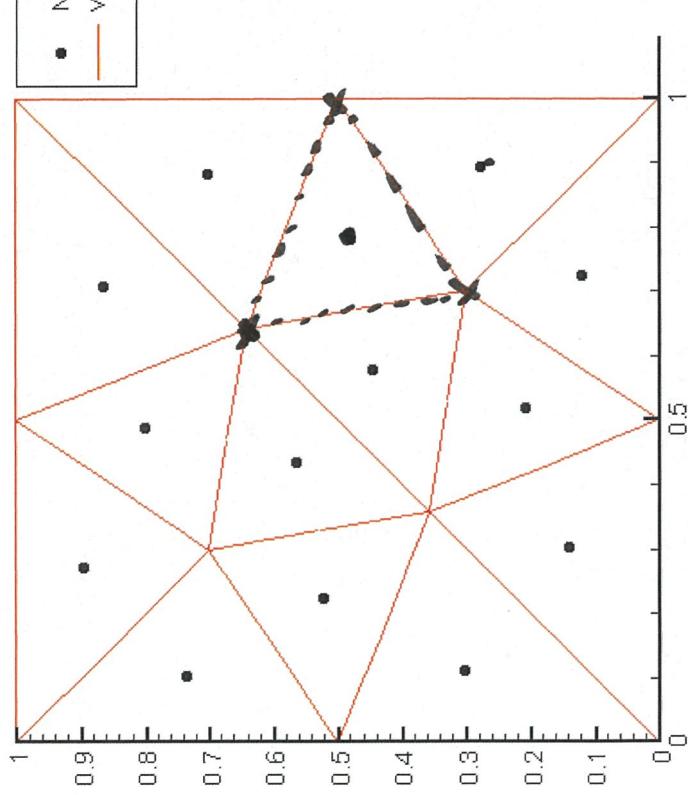
We also require an approximation for the integral of $\mathbf{J} \cdot \hat{\mathbf{n}}$ over the control volume surface ∂V_i .

Much of the rest of this chapter will be spent addressing how to do this appropriately.

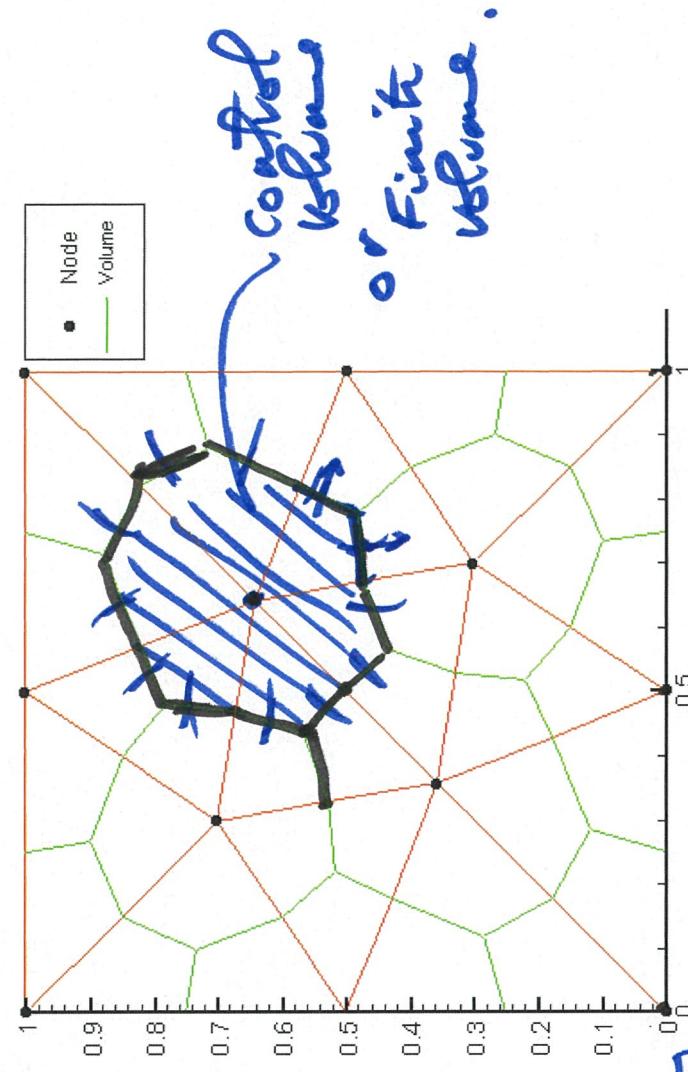


Illustration of Control Volume Construction in 2D

cell-centred



(a)



(b)

Figure: Constructing control volumes in two dimensions: (a) cell-centred; (b) vertex-centred.



Notation in one dimension

The one-dimensional version of the general transport equation (1) for a problem with a constant value of ρ and where $\mathbf{v} = u \mathbf{i}$ with u, S_φ and Γ potentially functions of the dependent variable φ , is expressed as

$$\rightarrow \frac{\partial \varphi}{\partial t} + \frac{\partial}{\partial x} \left(u \varphi - D \frac{\partial \varphi}{\partial x} \underbrace{\quad}_{J} \right) = S \quad (4)$$

where we have divided throughout by ρ and set $D = \Gamma / \rho$ and $S = S_\varphi / \rho$.

In (4) the i^{th} component of the flux vector J is given by $J = u \varphi - D \frac{\partial \varphi}{\partial x}$.

$$\boxed{\frac{\partial \varphi}{\partial t} + \frac{\partial J}{\partial x} = S}, \quad \overline{J}$$



When working in one dimension, we adopt the notation illustrated in the figure for a particular control volume of interest. Extending the control volume faces in the vertical direction is done purely for illustrative purposes: these are one-dimensional structures.

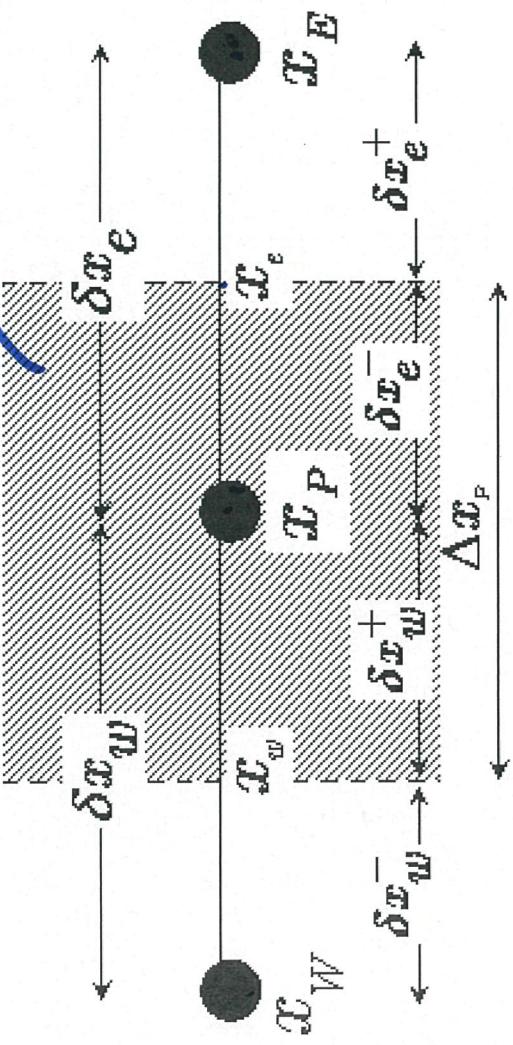


Figure: Control volume in one dimension

Illustration of Cell-Centred & Vertex-Centred Schemes in 1D

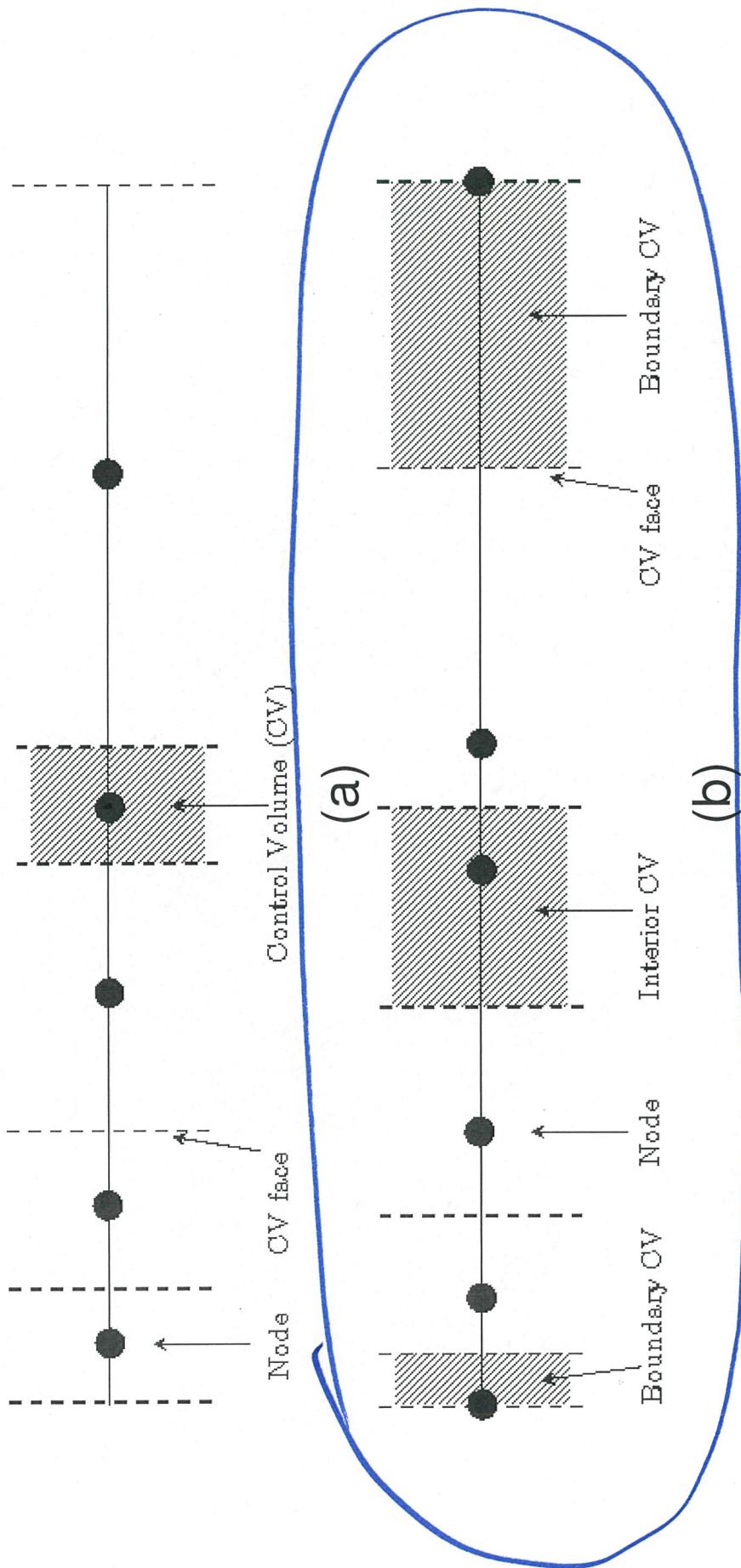


Figure: Constructing control volumes in one dimension: (a) cell-centred; (b) vertex-centred.

Diffusion Equation

We begin by considering the linear diffusion equation with source term on the interval $[0, L]$:

Diffusivity

$$\frac{\partial \varphi}{\partial t} + \frac{\partial}{\partial x} \left(-D \frac{\partial \varphi}{\partial x} \right) = S, \quad 0 < x < L, \quad t > 0 \quad (5)$$

$$\begin{aligned} & \frac{\partial}{\partial x} \frac{\partial \varphi}{\partial x} = \frac{A_0 \varphi - A_* \varphi}{B_* K} \\ & \downarrow \\ & \text{subject to} \end{aligned}$$

$$\begin{aligned} \text{at } x = 0, \quad A_0 \varphi - B_0 \frac{\partial \varphi}{\partial x} &= C_0, \quad t > 0 \\ \text{at } x = L, \quad A_L \varphi + B_L \frac{\partial \varphi}{\partial x} &= C_L, \quad t > 0 \end{aligned} \quad (6)$$

and initially

$$\varphi(x, 0) = \varphi_0(x), \quad 0 \leq x \leq L.$$



$$\rho C_p \frac{\partial \theta}{\partial t} = \left[\frac{k}{pc_1} \right] \left[\frac{\partial^2 \theta}{\partial x^2} \right]$$

thermal
diffusivity.

Spatial Discretisation

First we integrate from x_w to x_e to obtain

$$\int_{x_w}^{x_e} \frac{\partial \varphi}{\partial t} dx + \int_{x_w}^{x_e} \frac{\partial J}{\partial x} dx = \int_{x_w}^{x_e} S dx.$$

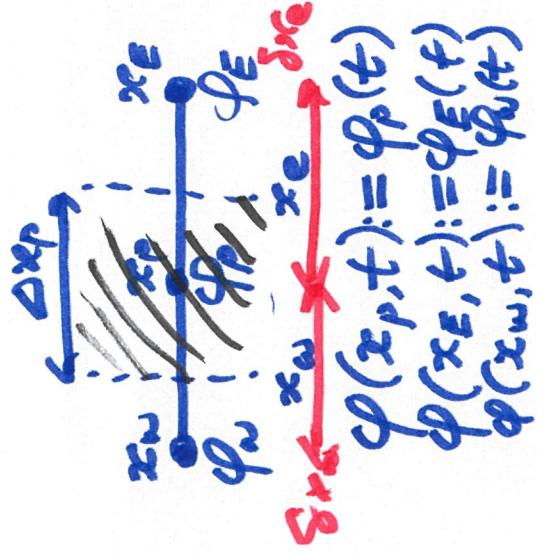
Then, proceeding as in the previous section, we obtain

$$\frac{d\varphi_P}{dt} = \frac{1}{\Delta x_P} (J_w - J_e) + S_P. \quad (7)$$

However, in order to apply this equation, we need expressions for J_w and J_e : the fluxes evaluated at the west and east face respectively.

We approximate $\left(\frac{\partial \varphi}{\partial x}\right)_w$ by $\frac{\varphi_P - \varphi_W}{\delta x_w}$ and $\left(\frac{\partial \varphi}{\partial x}\right)_e$ by $\frac{\varphi_E - \varphi_P}{\delta x_e}$, and hence

$$J_w = \left(-D \frac{\partial \varphi}{\partial x}\right)_w \approx -D \left(\frac{\varphi_P - \varphi_W}{\delta x_w}\right), \quad J_e = \left(-D \frac{\partial \varphi}{\partial x}\right)_e \approx -D \left(\frac{\varphi_E - \varphi_P}{\delta x_e}\right). \quad (8)$$



$$\begin{aligned}
 \frac{d}{dt} \int_{x_0}^{x_e} \varphi(z, t) dx + \{ J(x_0, t) - J(x_e, t) \} &= \int_{x_0}^{x_e} S(z, t) dx \\
 \bar{\varphi}_p &= \frac{1}{Dx_p} \int_{x_0}^{x_e} \varphi(z, t) dx ; \quad \bar{S}_p = \frac{1}{\Delta x_p} \int_{x_0}^{x_e} S(z, t) dx \\
 \frac{d\bar{\varphi}_p}{dt} + \frac{1}{\Delta x_p} \left\{ \bar{J}_e(t) - \bar{J}_0(t) \right\} &= \bar{S}_p \\
 \therefore \quad \dots &
 \end{aligned}$$

How should we approximate
 these terms?
 we approximate

IDEA:
 $\bar{\varphi}_p \approx \varphi_p$, $\bar{S}_p \approx S_p$
 $\bar{J}_e = [D \frac{\partial \varphi}{\partial x}]_{x=x_e}$, $\bar{J}_0 = [-D \frac{\partial S}{\partial x}]_{x=x_0}$
 $J_e \approx -D_e \frac{(\varphi_e - \varphi_r)}{\delta x_e}$, $J_0 \approx -D_0 \frac{(\varphi_p - \varphi_u)}{\delta x_u}$

Substitution gives :

$$\frac{d\varphi_p}{dt} = - \frac{1}{\Delta x_p} \left\{ -\frac{D_e}{\delta x_e} (\varphi_E(t) - \varphi_p(t)) + \frac{D_u}{\delta x_e} (\varphi_u(t) - \varphi_p(t)) \right\}$$

Rearranging : (constant diffusivity)

$$\frac{d\varphi_p}{dt} = \frac{D}{\Delta x_p} \left[\frac{1}{\delta x_e} \varphi_E(t) - \left(\frac{1}{\delta x_e} + \frac{1}{\delta x_u} \right) \varphi_p(t) + \frac{1}{\delta x_u} \varphi_u(t) \right] + S_p$$

- Finite volume eqⁿ (FVE)
Volume fraction solution eqⁿ.

Substituting (8) into (7) we obtain, after rearrangement,

$$\frac{d\varphi_P}{dt} = \frac{D}{\Delta x_P} \left[\left(\frac{1}{\delta x_W} \right) \varphi_W - \left(\frac{1}{\delta x_W} + \frac{1}{\delta x_E} \right) \varphi_P + \left(\frac{1}{\delta x_E} \right) \varphi_E \right] + S_P \quad (9)$$

which is the **finite volume discretisation** of (5).

Note

Note that this is a discretisation in space only: the derivative in time remains.