

# Project 3

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AERO 4630-002  
Aerospace Structural Dynamics

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## Part 1a: Non-dimensionalization and Weak Form

The governing equation

$$\text{div} \boldsymbol{\sigma} + \mathbf{f} = \rho \ddot{\mathbf{u}} \quad (1)$$

can be put in its weak form. By redefining the second derivative and converting to index notation, the governing equation becomes

$$\sigma_{ij,j} + f_i = \frac{\rho(u_i^{n+1} - 2u_i^n + u_i^{n-1}))}{(\Delta t)^2} \quad (2)$$

which can be rearranged to

$$\frac{1}{\rho} \sigma_{ij,j} (\Delta t)^2 + \frac{1}{\rho} f_i (\Delta t)^2 = u_i^{n+1} - 2u_i^n + u_i^{n-1} \quad (3)$$

By integrating over the body  $\Omega$ , the integral becomes

$$\int_{\Omega} \frac{1}{\rho} \sigma_{ij,j} (\Delta t)^2 dV + \int_{\Omega} \frac{1}{\rho} f_i (\Delta t)^2 dV = \int_{\Omega} (u_i^{n+1} - 2u_i^n + u_i^{n-1}) dV \quad (4)$$

We can multiply by a perturbation  $v_i$  and apply the chain rule to the first term.

$$\int_{\Omega} \frac{(\Delta t)^2}{\rho} \sigma_{ij,j} v_i dV = \int_{\Omega} \frac{(\Delta t)^2}{\rho} (\sigma_{ij} v_i)_{,j} dV - \int_{\Omega} \frac{(\Delta t)^2}{\rho} \sigma_{ij} v_{i,j} dV \quad (5)$$

Applying the divergence theorem yields

$$\int_{\Omega} \frac{(\Delta t)^2}{\rho} \sigma_{ij,j} v_i dV = \int_{\partial\Omega} \frac{(\Delta t)^2}{\rho} \sigma_{ij} n_j v_i dA - \int_{\Omega} \frac{(\Delta t)^2}{\rho} \sigma_{ij} v_{i,j} dV \quad (6)$$

The governing equation in its weak form becomes

$$-\int_{\Omega} \frac{(\Delta t)^2}{\rho} \sigma_{ij}(\mathbf{u}^{n+1}) v_i dV + \int_{\partial\Omega} \frac{(\Delta t)^2}{\rho} t_i v_i dA + \int_{\Omega} \frac{(\Delta t)^2}{\rho} f_i v_i dV = \int_{\Omega} v_i (u_i^{n+1} - 2u_i^n + u_i^{n-1}) dV \quad (7)$$

Length terms, traction and stress terms, and time terms in the weak form equation can be nondimensionalized by the beam length  $L$ , Young's modulus  $E$ , and characteristic time  $\tilde{t}$ .

$$-\int_{\Omega} \frac{(\Delta t)^2}{\rho(\tilde{t})^2} \frac{\sigma_{ij}(\mathbf{u}^{n+1}) v_i}{EL^2} dV + \int_{\partial\Omega} \frac{(\Delta t)^2}{\rho(\tilde{t})^2} \frac{t_i v_i}{EL} dA + \int_{\Omega} \frac{(\Delta t)^2}{\rho(\tilde{t})^2} \frac{f_i v_i}{L} dV = \int_{\Omega} \frac{v_i (u_i^{n+1} - 2u_i^n + u_i^{n-1})}{L^2} dV \quad (8)$$

## Part 1b: Just Displacement

Deflection was determined at the center of a beam with length  $L = 1\text{m}$ , width  $W = 0.2\text{m}$ , height  $H = 0.2\text{m}$ , Young's Modulus  $E = 200\text{GPa}$ , Poisson ratio  $\nu = 0.3$ , and density  $7800\text{kg m}^{-3}$ . The beam is clamped at both ends, and a downward force of  $F = 100\text{N}$  is applied over the area on the top surface near the center, within  $0.2\text{ m}$  along the width, and within  $0.02\text{ m}$  along the length of the beam.

The deflection was determined to be  $3.75 \times 10^{-8}\text{ m}$  downward, and the Paraview output is shown in Fig 1

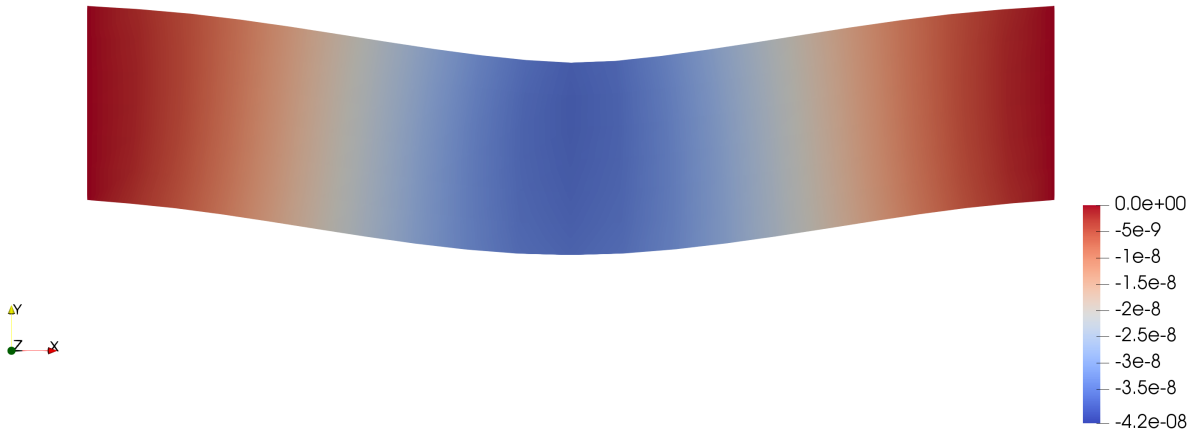


Figure 1: Paraview output for problem 1b

The code used for part 1b can be seen in **Appendix 1b**.

## Part 1c: Free Vibration

The force was removed, and the calculated deflection was set as an initial condition of the beam. A free vibration was modeled, and a time-plot can be seen in Fig 2.

The period of oscillation was determined to be 0.0011 seconds, and the natural frequency was determined to be 5681 rad/s or 904.2 Hz.

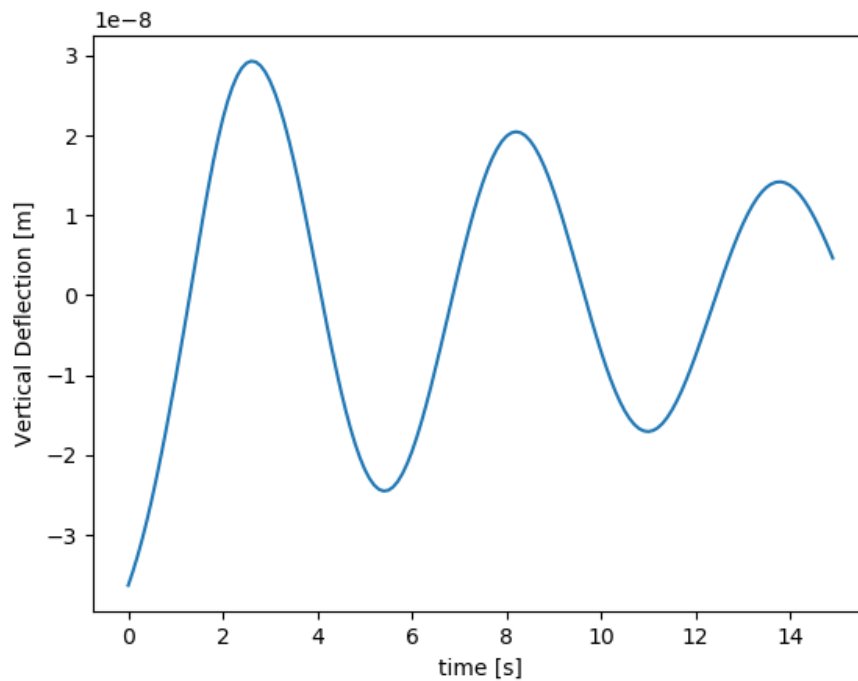


Figure 2: Vertical Deflection vs Time

The code used for Problem 1c is shown in **Appendix 1c**.

## Part 1d: Natural Frequency, Changing Dimensions

### Changing Width

Next, the beam width was varied to assess how natural frequency changes with beam width. Widths of 0.2, 0.4, 0.6, 0.8, and 1.0 m were assessed. A plot of natural frequencies over beam width is shown in Fig 3.

The Python code used for part 1d-i is shown in **Appendix 1d-i**.

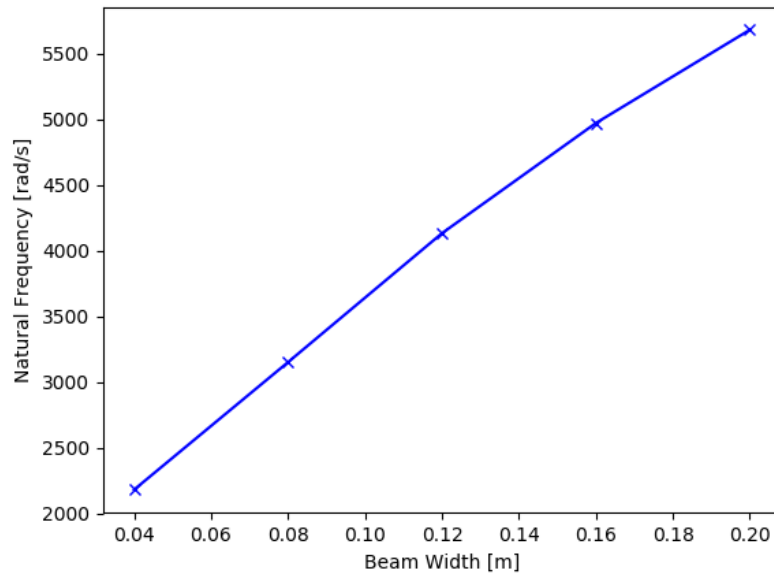


Figure 3: Natural Frequency over Beam Width for Problem 1d

### Changing Height

Next, the beam height was varied to assess how natural frequency changes with beam height. Heights of 0.2, 0.4, 0.6, 0.8, and 1.0 m were assessed. A plot of natural frequencies over beam Height is shown in Fig 4.

The Python code used for part 1d-ii is shown in **Appendix 1d-ii**.

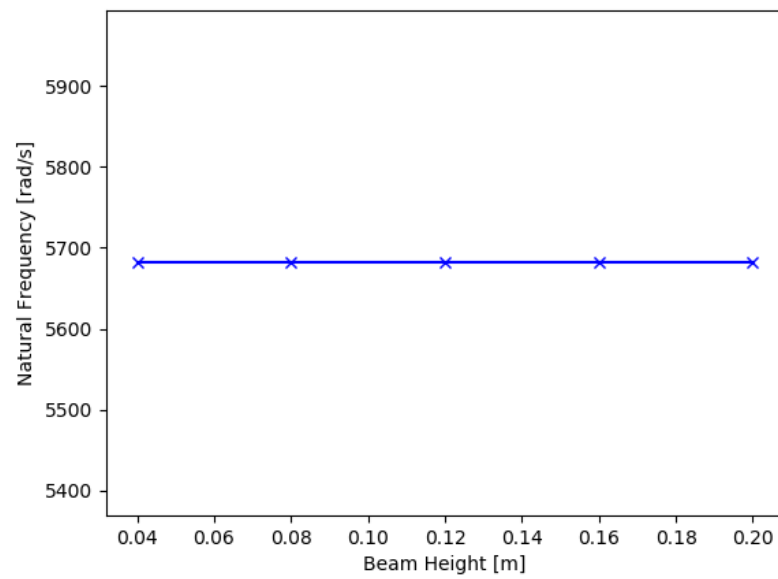


Figure 4: Natural Frequency over Beam Height for Problem 1d

## Appendix 1b: Code for Problem 1b

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"""

*Python script for Part 1b of Project 2a*

*Original Author: Vinamra Agrawal*

*Date: January 25, 2019*

*Edited By: Omkar Mulekar*

*Date: February 10, 2019*

"""

```
from __future__ import print_function
from fenics import *
import matplotlib
matplotlib.use("Agg")
import matplotlib.pyplot as plt
from ufl import nabla_div
import math
```

```
##=====
# Define System Properties
#=====
```

```
length = 1;
W = 0.2;
H = 0.2;
```

```
a = 0.04/length;
b = 0.4*H/length;
area = a*b;
```

```
F = -100
```

```
youngs = 200e9 # Youngs
nu = 0.3 # Poisson
rho = 7800 # Density
```

```
# Lamé parameters
mu = (youngs)/(2*(1+nu))
lambda_ = (nu*youngs)/((1+nu)*(1-2*nu))
```

```
g = 10
```

```

traction_applied = F/area

#=====
# Dimensionless parameters
#=====
l_nd = length/length
w_nd = W/length
h_nd = H/length

mu_nd = mu/youngs
lambda_nd = lambda_/youngs

traction_nd = traction_applied/youngs

#=====
# Boundaries and Geometry
#=====
mesh = BoxMesh(Point(0,0,0),Point(l_nd,w_nd,h_nd),20,6,6)
V = VectorFunctionSpace(mesh,'P',1)

tol = 1E-14

def boundary_left(x,on_boundary):
    return (on_boundary and near(x[0],0,tol))

def boundary_right(x,on_boundary):
    return on_boundary and near(x[0],l_nd,tol)

bc_left = DirichletBC(V,Constant((0,0,0)),boundary_left)
bc_right = DirichletBC(V,Constant((0,0,0)),boundary_right)

#=====
def epsilon(u):
    return 0.5*(nabla_grad(u) + nabla_grad(u).T)

def sigma(u):
    return lambda_nd*nabla_div(u)*Identity(d) + mu_nd*(epsilon(u) +
        epsilon(u).T)

#=====
# First we solve the problem of a cantilever beam under fixed
# load.
#=====

```

```

u_init = TrialFunction(V)
d = u_init.geometric_dimension()
v = TestFunction(V)
f = Constant((0.0,0.0,0.0))
T_init = Expression(('0.0', 'x[0] >= 0.48*l && x[0] <= .52*l &&
    near(x[1],w) && x[2] >= 0.3*h && x[2] <= 0.7*h? A : 0.0', '0.0',
    ), degree=1, l=l_nd, w=w_nd, h=h_nd, A=traction_nd)
F_init = inner(sigma(u_init),epsilon(v))*dx - dot(f,v)*dx - dot(
    T_init,v)*ds
a_init, L_init = lhs(F_init), rhs(F_init)

print("Solving the initial cantilever problem")
u_init = Function(V)
solve(a_init==L_init,u_init,[bc_left,bc_right])
w_nd = u_init(l_nd/2.0,w_nd/2.0,h_nd/2.0)
w = w_nd * length
print(w[1])

vtkfile_u = File('deflection.pvd')
vtkfile_u << u_init

```

---



## Appendix 1c: Code for Problem 1c

---

"""

*Python script for Part 1c of Project 2a*

*Original Author: Vinamra Agrawal*

*Date: January 25, 2019*

*Edited By: Omkar Mulekar*

*Date: February 28, 2019*

"""

```
from __future__ import print_function
from fenics import *
import matplotlib
matplotlib.use("Agg")
import matplotlib.pyplot as plt
from ufl import nabla_div
import math
import numpy as np
from scipy.signal import argrelextrema
```

```
##=====
# Define System Properties
#=====
```

```
length = 1;
W = 0.2;
H = 0.2;
```

```
a = 0.04*length;
b = 0.4*H;
area = a*b;
```

```
F = -100
```

```
youngs = 200e9 # Youngs
nu = 0.3 # Poisson
rho = 7800 # Density
```

```
# Lamé parameters
mu = (youngs)/(2*(1+nu))
lambda_ = (nu*youngs)/((1+nu)*(1-2*nu))
```

```

g = 10

traction_applied = F/area

#=====
# Dimensionless parameters
#=====
l_nd = length/length
w_nd = W/length
h_nd = H/length

bar_speed = math.sqrt(youngs/rho)
t_char = length/bar_speed
t = 0
t_i = 0.5
dt = 0.1
num_steps = 150

mu_nd = mu/youngs
lambda_nd = lambda_/youngs

traction_nd = traction_applied/youngs

#=====
# Boundaries and Geometry
#=====
mesh = BoxMesh(Point(0,0,0),Point(l_nd,w_nd,h_nd),20,6,6)
V = VectorFunctionSpace(mesh,'P',1)

tol = 1E-14

def boundary_left(x,on_boundary):
    return (on_boundary and near(x[0],0,tol))

def boundary_right(x,on_boundary):
    return on_boundary and near(x[0],l_nd,tol)

bc_left = DirichletBC(V,Constant((0,0,0)),boundary_left)
bc_right = DirichletBC(V,Constant((0,0,0)),boundary_right)

#=====
def epsilon(u):
    return 0.5*(nabla_grad(u) + nabla_grad(u).T)

```

```

def sigma(u):
    return lambda_nd*nabla_div(u)*Identity(d) + mu_nd*(epsilon(u) +
        epsilon(u).T)

=====
# First we solve the problem of a cantilever beam under fixed
# load.
=====
u_init = TrialFunction(V)
d = u_init.geometric_dimension()
v = TestFunction(V)
f = Constant((0.0,0.0,0.0))
T_init = Expression(('0.0', 'x[0] >= 0.48*l && x[0] <= .52*l &&
    near(x[1],w) && x[2] >= 0.3*h && x[2] <= 0.7*h? A : 0.0', '0.0',
    ), degree=1, l=l_nd, w=w_nd, h=h_nd, A=traction_nd)
F_init = inner(sigma(u_init),epsilon(v))*dx - dot(f,v)*dx - dot(
    T_init,v)*ds
a_init, L_init = lhs(F_init), rhs(F_init)

print("Solving the initial cantilever problem")
u_init = Function(V)
solve(a_init==L_init, u_init, [bc_left, bc_right])
w_nd = u_init(l_nd/2.0, w_nd/2.0, h_nd/2.0)
w = w_nd * length
print(w[1])

=====
# Next we use this as initial condition, let the force go and
# study the vertical vibrations of the beam
=====
u_n = interpolate(Constant((0.0,0.0,0.0)),V)
u_n_1 = interpolate(Constant((0.0,0.0,0.0)),V)
u_n.assign(u_init)
u_n_1.assign(u_init)

T_n = Constant((0.0,0.0,0.0))

u = TrialFunction(V)
d = u.geometric_dimension()
v = TestFunction(V)

F = (dt*dt)*inner(sigma(u),epsilon(v))*dx + dot(u,v)*dx - (dt*dt)*
    dot(f,v)*dx - (dt*dt)*dot(T_n,v)*ds - 2.0*dot(u_n,v)*dx + dot(
    u_n_1,v)*dx

```

```

a,L = lhs(F) , rhs(F)

xdmffile_u = XDMFFile( 'results/solution.xdmf' )
xdmffile_s = XDMFFile( 'results/stress.xdmf' )

u = Function(V)

u_store = [0] * num_steps
time = [0] * num_steps

index = 0
for n in range(num_steps):
    print("time = %.2f" % t)
    T_n.t = t
    solve(a == L, u, [bc_left , bc_right])
    u_grab = u(0.5,0.1,0.1)
    u_store[n] = u_grab[1]

    if(abs(t-index)<0.01):
        print("Writing output files...")
        xdmffile_u.write(u*length,t)
        W = TensorFunctionSpace(mesh, "Lagrange", 1)
        stress = lambda_*nabla_div(u)*Identity(d) + mu*(epsilon(u)
            + epsilon(u).T)
        xdmffile_s.write(project(stress,W),t)
        index += 1

    time[n] = t
    t+=dt
    u_n_1.assign(u_n)
    u_n.assign(u)

# Get period of oscillation
u_np = np.array(u_store)
min_args = argrelextrema(u_np,np.less)
period = (time[min_args[0][1]] - time[min_args[0][0]])*t_char
nat_freq = 2*math.pi /period
print("Period of Oscillation", period, " seconds")
print("Natural Frequency:    ", nat_freq," rad/s")

plt.figure(1)
plt.plot(time,u_store)
plt.xlabel('time [s]')
plt.ylabel('Vertical Deflection [m]')

```

```
plt.savefig('1cfig.png')
```

---

## Appendix 1d-i: Code for Problem 1d-i

---

"""

*Python script for Part 1d.i of Project 2a*

*Original Author: Vinamra Agrawal*

*Date: January 25, 2019*

*Edited By: Omkar Mulekar*

*Date: February 28, 2019*

"""

```
from __future__ import print_function
from fenics import *
import matplotlib
matplotlib.use("Agg")
import matplotlib.pyplot as plt
from ufl import nabla_div
import math
import numpy as np
from scipy.signal import argrelextrema
```

```
##=====
# Define System Properties
#=====
length = 1;
W1 = 0.2;
H = 0.2;
alpha = np.linspace(0.2,1,num=5)
print("alpha = ",alpha)
W = alpha * W1

a = 0.04*length;
b = 0.4*H;
area = a*b;

F = -100

youngs = 200e9 # Youngs
nu = 0.3 # Poisson
rho = 7800 # Density
```

```

# Lame parameters
mu = (youngs)/(2*(1+nu))
lambda_ = (nu*youngs)/((1+nu)*(1-2*nu))

g = 10

traction_applied = F/area

nat_freq = [0] * len(alpha)

print("Beginning Loop...")
for i in range(len(alpha)):
    print("alpha = ", alpha[i])

#=====
# Dimensionless parameters
#=====
l_nd = length/length
w_nd = W[i]/length
h_nd = H/length

bar_speed = math.sqrt(youngs/rho)
t_char = length/bar_speed
t = 0
t_i = 0.5
dt = 0.1
num_steps = 275

mu_nd = mu/youngs
lambda_nd = lambda_/youngs

traction_nd = traction_applied/youngs

#=====
# Boundaries and Geometry
#=====
mesh = BoxMesh(Point(0,0,0),Point(l_nd,w_nd,h_nd),20,6,6)
V = VectorFunctionSpace(mesh,'P',1)

tol = 1E-14

def boundary_left(x,on_boundary):
    return (on_boundary and near(x[0],0,tol))

```

```

def boundary_right(x,on_boundary):
    return on_boundary and near(x[0],l_nd,tol)

bc_left = DirichletBC(V,Constant((0,0,0)),boundary_left)
bc_right = DirichletBC(V,Constant((0,0,0)),boundary_right)

=====
def epsilon(u):
    return 0.5*(nabla_grad(u) + nabla_grad(u).T)

def sigma(u):
    return lambda_nd*nabla_div(u)*Identity(d) + mu_nd*(epsilon(u)
        ) + epsilon(u).T)

=====
# First we solve the problem of a cantilever beam under fixed
# load.
=====
u_init = TrialFunction(V)
d = u_init.geometric_dimension()
v = TestFunction(V)
f = Constant((0.0,0.0,0.0))
T_init = Expression(('0.0', 'x[0] >= 0.48*l && x[0] <= .52*l &&
    near(x[1],w) && x[2] >= 0.3*h && x[2] <= 0.7*h? A : 0.0', '
    0.0'), degree=1, l=l_nd, w=w_nd, h=h_nd, A=traction_nd)
F_init = inner(sigma(u_init),epsilon(v))*dx - dot(f,v)*dx - dot
    (T_init,v)*ds
a_init, L_init = lhs(F_init), rhs(F_init)

print("Solving the initial cantilever problem")
u_init = Function(V)
solve(a_init==L_init,u_init,[bc_left,bc_right])
down_nd = u_init(l_nd/2.0,w_nd/2.0,h_nd/2.0)
w = down_nd * length
print("Initial Displacement: ",w[1])

=====
# Next we use this as initial condition, let the force go and
# study the vertical vibrations of the beam
=====
u_n = interpolate(Constant((0.0,0.0,0.0)),V)
u_n_1 = interpolate(Constant((0.0,0.0,0.0)),V)
u_n.assign(u_init)
u_n_1.assign(u_init)

```



```

T_n = Constant((0.0,0.0,0.0))

u = TrialFunction(V)
d = u.geometric_dimension()
v = TestFunction(V)

F = (dt*dt)*inner(sigma(u),epsilon(v))*dx + dot(u,v)*dx - (dt*
    dt)*dot(f,v)*dx - (dt*dt)*dot(T_n,v)*ds - 2.0*dot(u_n,v)*dx
    + dot(u_n_1,v)*dx
a,L = lhs(F), rhs(F)

u_store = [0] * num_steps
time = [0] * num_steps

index = 0
for n in range(num_steps):
    print("time = %.2f" % t)
    T_n.t = t
    u = Function(V)
    solve(a == L, u, [bc_left, bc_right])
    u_grab = u(l_nd/2.0, w_nd/2.0, h_nd/2.0)
    u_store[n] = u_grab[1]

    if(abs(t-index)<0.01):
        # print("Writing output files...")
        W_ = TensorFunctionSpace(mesh, "Lagrange", 1)
        stress = lambda_*nabla_div(u)*Identity(d) + mu*(epsilon(
            u) + epsilon(u).T)
        index += 1

    time[n] = t
    t+=dt
    u_n_1.assign(u_n)
    u_n.assign(u)

plt.figure(1)
plt.plot(time, u_store)
plt.xlabel('time [s]')
plt.ylabel('Vertical Deflection [m]')
plt.savefig('1dfig_test.png')

# Get period of oscillation
u_np = np.array(u_store)

```

```

min_args = argrelextrema(u_np,np.greater)
print("min_args", min_args[0])
period = (time[min_args[0][1]] - time[min_args[0][0]])*t_char
nat_freq[i] = 2*math.pi /period
print("Period of Oscillation", period, " seconds")
print("Natural Frequency:    ", nat_freq," rad/s")

plt.figure(2)
plt.plot(W,nat_freq,'b-x')
plt.xlabel('Beam Width [m]')
plt.ylabel('Natural Frequency [rad/s]')
plt.savefig('1dfig_alpha.png')

```

---

## Appendix 1d-ii: Code for Problem 1d-ii

---

"""

*Python script for Part 1d.ii of Project 2a*

*Original Author: Vinamra Agrawal*

*Date: January 25, 2019*

*Edited By: Omkar Mulekar*

*Date: February 28, 2019*

"""

```
from __future__ import print_function
from fenics import *
import matplotlib
matplotlib.use("Agg")
import matplotlib.pyplot as plt
from ufl import nabla_div
import math
import numpy as np
from scipy.signal import argrelextrema
```

```
##=====
# Define System Properties
#=====
length = 1;
W = 0.2;
H1 = 0.2;
beta = np.linspace(0.2,1,num=5)
print("beta = ", beta)
H = beta * H1
print("H = ", H)
```

```
youngs = 200e9 # Youngs
nu = 0.3 # Poisson
rho = 7800 # Density
```

```
# Lamé parameters
mu = (youngs)/(2*(1+nu))
```

```
lambda_ = (nu*youngs)/((1+nu)*(1-2*nu))
```

```
g = 10
```

```
nat_freq = [0] * len(beta)
```

```
print("Beginning Loop...")
```

```
for i in range(len(beta)):
```

```
    print("beta = ", beta[i])
```

```
##
```

```
# Dimensionless parameters
```

```
##
```

```
l_nd = length/length
```

```
w_nd = W/length
```

```
h_nd = H[i]/length
```

```
bar_speed = math.sqrt(youngs/rho)
```

```
t_char = length/bar_speed
```

```
t = 0
```

```
t_i = 0.5
```

```
dt = 0.1
```

```
num_steps = 275
```

```
mu_nd = mu/youngs
```

```
lambda_nd = lambda_/youngs
```

```
F = -100
```

```
a = 0.04*length;
```

```
b = 0.4*H[i];
```

```
traction_applied = F/(a*b)
```

```
traction_nd = traction_applied/youngs
```

```
##
```

```
# Boundaries and Geometry
```

```
##
```

```
mesh = BoxMesh(Point(0,0,0),Point(l_nd,w_nd,h_nd),20,6,6)
```

```
V = VectorFunctionSpace(mesh,'P',1)
```

```
tol = 1E-14
```

```
def boundary_left(x,on_boundary):
```

```
    return (on_boundary and near(x[0],0,tol))
```

```

def boundary_right(x,on_boundary):
    return on_boundary and near(x[0],l_nd,tol)

bc_left = DirichletBC(V,Constant((0,0,0)),boundary_left)
bc_right = DirichletBC(V,Constant((0,0,0)),boundary_right)

=====
def epsilon(u):
    return 0.5*(nabla_grad(u) + nabla_grad(u).T)

def sigma(u):
    return lambda_nd*nabla_div(u)*Identity(d) + mu_nd*(epsilon(u)
        + epsilon(u).T)

=====
# First we solve the problem of a cantelever beam under fixed
# load.
=====
u_init = TrialFunction(V)
d = u_init.geometric_dimension()
v = TestFunction(V)
f = Constant((0.0,0.0,0.0))
T_init = Expression(('0.0', 'x[0] >= 0.48*l && x[0] <= .52*l &&
    near(x[1],w) && x[2] >= 0.3*h && x[2] <= 0.7*h? A : 0.0', '
    0.0'), degree=1, l=l_nd, w=w_nd, h=h_nd, A=traction_nd)
F_init = inner(sigma(u_init),epsilon(v))*dx - dot(f,v)*dx - dot
    (T_init,v)*ds
a_init, L_init = lhs(F_init), rhs(F_init)

print("Solving the initial cantelever problem")
u_init = Function(V)
solve(a_init==L_init, u_init, [bc_left, bc_right])
down_nd = u_init(l_nd/2.0,w_nd/2.0,h_nd/2.0)
w = down_nd * length
print("Initial Displacement: ",w[1])

=====
# Next we use this as initial condition, let the force go and
# study the vertical vibrations of the beam
=====
u_n = interpolate(Constant((0.0,0.0,0.0)),V)
u_n_1 = interpolate(Constant((0.0,0.0,0.0)),V)
u_n.assign(u_init)

```

```

u_n_1.assign(u_init)

T_n = Constant((0.0,0.0,0.0))

u = TrialFunction(V)
d = u.geometric_dimension()
v = TestFunction(V)

F = (dt*dt)*inner(sigma(u),epsilon(v))*dx + dot(u,v)*dx - (dt*
    dt)*dot(f,v)*dx - (dt*dt)*dot (T_n,v)*ds - 2.0*dot(u_n,v)*dx
    + dot(u_n_1,v)*dx
a,L = lhs(F), rhs(F)

u_store = [0] * num_steps
time = [0] * num_steps

index = 0
for n in range(num_steps):
    print("time = %.2f" % t)
    T_n.t = t
    u = Function(V)
    solve(a == L, u, [bc_left, bc_right])
    u_grab = u(l_nd/2.0,w_nd/2.0,h_nd/2.0)
    u_store[n] = u_grab[1]

    if(abs(t-index)<0.01):
        W_ = TensorFunctionSpace(mesh, "Lagrange", 1)
        stress = lambda_*nabla_div(u)*Identity(d) + mu*(epsilon(
            u) + epsilon(u).T)
        index += 1

    time[n] = t
    t+=dt
    u_n_1.assign(u_n)
    u_n.assign(u)

plt.figure(1)
plt.plot(time,u_store)
plt.xlabel('time [s]')
plt.ylabel('Vertical Deflection [m]')
plt.savefig('1dfig_test2.png')

# Get period of oscillation
u_np = np.array(u_store)

```

```

min_args = argrelextrema(u_np,np.greater)
print("min_args", min_args[0])
period = (time[min_args[0][1]] - time[min_args[0][0]])*t_char
nat_freq[i] = 2*math.pi /period
print("Period of Oscillation", period, " seconds")
print("Natural Frequency:    ", nat_freq," rad/s")

plt.figure(2)
plt.plot(H,nat_freq,'b-x')
plt.xlabel('Beam Height [m]')
plt.ylabel('Natural Frequency [rad/s]')
plt.savefig('1dfig_beta.png')

```

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