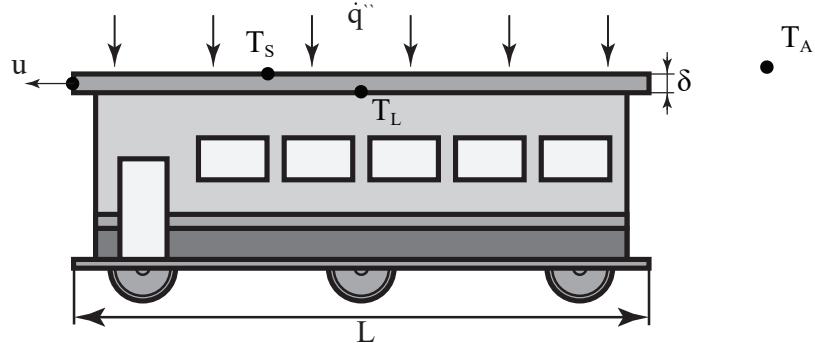


Exercise III.6: (Moving train ★)

Consider the roof surface of a passenger car on a moving train. This surface is exposed to solar radiation, with an incident heat flux denoted as \dot{q}'' . The ambient air temperature is represented by T_A .

**Given parameters:**

- Velocity of the train: $u = 50 \text{ km/h}$
- Length of the train roof: $L = 10 \text{ m}$
- Width of the train roof: $W = 3 \text{ m}$
- Thickness of the train roof: $\delta = 20 \text{ cm}$
- Thermal conductivity of the train roof: $\lambda = 0.03 \text{ W/mK}$
- Lower temperature of the train roof: $T_L = 16 \text{ }^\circ\text{C}$
- Ambient temperature: $T_A = 15 \text{ }^\circ\text{C}$
- Solar irradiation: $\dot{q}'' = 288 \text{ W/m}^2$
- Properties of air:

| T ($^\circ\text{C}$) | ρ (kg/m^3) | c ($\text{kJ/kg} \cdot \text{K}$) | λ (W/mK) | ν (m^2/s) | Pr |
|-----------------------------|-------------------------------|--|--------------------------------|------------------------------------|--------|
| 0 | 1.275 | 1.006 | $24.18 \cdot 10^{-3}$ | $13.52 \cdot 10^{-6}$ | 0.7179 |
| 20 | 1.188 | 1.007 | $25.69 \cdot 10^{-3}$ | $15.35 \cdot 10^{-6}$ | 0.7148 |
| 40 | 1.112 | 1.007 | $27.16 \cdot 10^{-3}$ | $17.26 \cdot 10^{-6}$ | 0.7122 |
| 80 | 0.9859 | 1.008 | $30.01 \cdot 10^{-3}$ | $21.35 \cdot 10^{-6}$ | 0.7083 |
| 100 | 0.9329 | 1.009 | $31.39 \cdot 10^{-3}$ | $23.51 \cdot 10^{-6}$ | 0.7073 |

Hints:

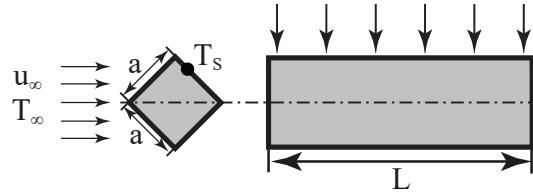
- Assume steady-state heat transfer to be one-dimensional
- Neglect radiation heat exchange with the surroundings.

Tasks:

- Determine the equilibrium temperature of the top surface T_s .

Exercise III.7: (Transverse flow ★★)

Air flows transversely across a beam of length L , with a square cross-sectional area, as can be seen in the figure. The circles with crosses indicate streamlines that are moving away from the observer.

**Given parameters:**

- Beam geometrical dimensions: a, L
- Material properties of the air: $\eta, \rho, \text{Pr}, \lambda$
- Velocity of the crossflow: u_∞
- Temperatures of the surface and ambient: T_s, T_∞

Hints:

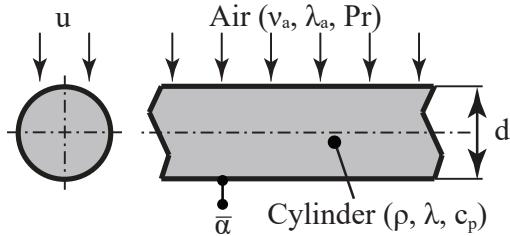
- Assume steady-state conditions
- $10^4 \leq \text{Re} \leq 10^5$
- Heat loss from the sides is negligible.

Tasks:

- Provide an expression for the rate of heat loss in terms of the given parameters.
- Determine the percentual change of the heat transfer coefficient if we had a similar-form rod with four times the crosswise width at double flow velocity.

Exercise III.8: (Heating of a cylinder **)

A long cylinder is kept at a homogeneous temperature T_0 . This cylinder is suddenly, at time $t = 0$, exposed to a warm air flow with temperature T_a and a transverse velocity of u .

**Given parameters:**

- Diameter of the cylinder: $d = 0.055 \text{ m}$
- Density of the cylinder: $\rho = 1500 \text{ kg/m}^3$
- Thermal conductivity of the cylinder: $\lambda = 0.119 \text{ W/mK}$
- Heat capacity of the cylinder: $c_p = 1000 \text{ J/kgK}$
- Initial homogenous temperature of the cylinder: $T_0 = 10 \text{ }^\circ\text{C}$
- Critical temperature of the cylinder: $T_c = 38 \text{ }^\circ\text{C}$
- Temperature of the air flow: $T_a = 40 \text{ }^\circ\text{C}$
- Velocity of the air: $u = 0.1 \text{ m/s}$
- Thermal conductivity of the air: $\lambda_a = 25.7 \cdot 10^{-3} \text{ W/mK}$
- Kinematic viscosity of the air: $\nu_a = 15.35 \cdot 10^{-6} \text{ m}^2/\text{s}$
- Prandtl number of the air: $\text{Pr} = 0.71$

Hints:

- The average heat transfer coefficient $\bar{\alpha}$ is steady in time.
- The material properties can be taken as constant.

Tasks:

- a) Determine the initial rate of heat transfer per unit length of the cylinder.
- b) At $t = t_c$ the cylinder reaches the critical temperature T_c at its hottest point. Sketch qualitatively the temperature distribution at time-points $t = 0$ and $t = t_c$.
- c) Determine the time t_c , until the cylinder reaches the critical temperature T_c at its hottest point.