

## Sample Problem 6/2

The vertical bar  $AB$  has a mass of 150 kg with center of mass  $G$  midway between the ends. The bar is elevated from rest at  $\theta = 0$  by means of the parallel links of negligible mass, with a constant couple  $M = 5 \text{ kN}\cdot\text{m}$  applied to the lower link at  $C$ . Determine the angular acceleration  $\alpha$  of the links as a function of  $\theta$  and find the force  $B$  in the link  $DB$  at the instant when  $\theta = 30^\circ$ .

**Solution.** The motion of the bar is seen to be curvilinear translation since the bar itself does not rotate during the motion. With the circular motion of the mass center  $G$ , we choose  $n$ - and  $t$ -coordinates as the most convenient description. With negligible mass of the links, the tangential component  $A_t$  of the force at  $A$  is obtained from the free-body diagram of  $AC$ , where  $\Sigma M_C \cong 0$  and

- ①  $A_t = M/AC = 5/1.5 = 3.33 \text{ kN}$ . The force at  $B$  is along the link. All applied forces are shown on the free-body diagram of the bar, and the kinetic diagram is also indicated, where the  $m\bar{a}$  resultant is shown in terms of its two components.

The sequence of solution is established by noting that  $A_n$  and  $B$  depend on the  $n$ -summation of forces and, hence, on  $m\bar{r}\omega^2$  at  $\theta = 30^\circ$ . The value of  $\omega$  depends on the variation of  $\alpha = \ddot{\theta}$  with  $\theta$ . This dependency is established from a force summation in the  $t$ -direction for a general value of  $\theta$ , where  $\bar{a}_t = (\bar{a}_t)_A = AC\alpha$ . Thus, we begin with

$$[\Sigma F_t = m\bar{a}_t] \quad 3.33 - 0.15(9.81) \cos \theta = 0.15(1.5\alpha)$$

$$\alpha = 14.81 - 6.54 \cos \theta \text{ rad/s}^2 \quad \text{Ans.}$$

With  $\alpha$  a known function of  $\theta$ , the angular velocity  $\omega$  of the links is obtained from

$$[\omega d\omega = \alpha d\theta] \quad \int_0^\omega \omega d\omega = \int_0^\theta (14.81 - 6.54 \cos \theta) d\theta$$

$$\omega^2 = 29.6\theta - 13.08 \sin \theta$$

Substitution of  $\theta = 30^\circ$  gives

$$(\omega^2)_{30^\circ} = 8.97 \text{ (rad/s)}^2 \quad \alpha_{30^\circ} = 9.15 \text{ rad/s}^2$$

and

$$m\bar{r}\omega^2 = 0.15(1.5)(8.97) = 2.02 \text{ kN}$$

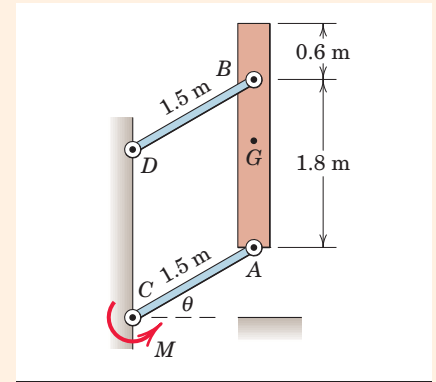
$$m\bar{r}\alpha = 0.15(1.5)(9.15) = 2.06 \text{ kN}$$

The force  $B$  may be obtained by a moment summation about  $A$ , which eliminates  $A_n$  and  $A_t$  and the weight. Or a moment summation may be taken about the intersection of  $A_n$  and the line of action of  $m\bar{r}\alpha$ , which eliminates  $A_n$  and  $m\bar{r}\alpha$ . Using  $A$  as a moment center gives

$$[\Sigma M_A = m\bar{a}d] \quad 1.8 \cos 30^\circ B = 2.02(1.2) \cos 30^\circ + 2.06(0.6)$$

$$B = 2.14 \text{ kN} \quad \text{Ans.}$$

The component  $A_n$  could be obtained from a force summation in the  $n$ -direction or from a moment summation about  $G$  or about the intersection of  $B$  and the line of action of  $m\bar{r}\alpha$ .



### Helpful Hints

- ① Generally speaking, the best choice of reference axes is to make them coincide with the directions in which the components of the mass-center acceleration are expressed. Examine the consequences of choosing horizontal and vertical axes.
- ② The force and moment equations for a body of negligible mass become the same as the equations of equilibrium. Link  $BD$ , therefore, acts as a two-force member in equilibrium.

