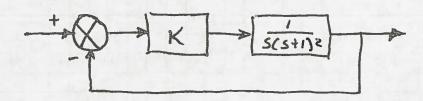
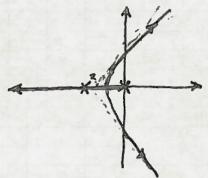
Consider the following system:



We can sketch the Root Locus and Know that increasing the gain will cause instability ...



(This happens at a gain K= Z in this system)

We know that roots of the characteristic equation satisfy

1+ KG(s) = 0

or, equivalently, that | KG(s) |= 1 and LG(s) = 180°

When the roots cross over the imaginary ax's, we know we have a purely imaginary root. Call this value of s=jw.

Then we have |KGijws|=1 and KGijw)=180°.

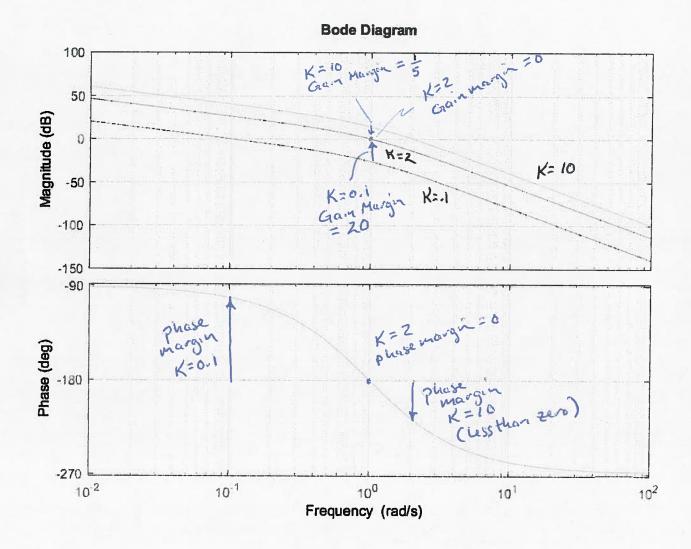
We can see these conditions for neutral stability directly from the Bode Plot. It tells us how much we can increase the gain before neutral stability.

We can define a

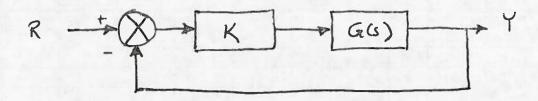
Grain margin - How much we can further increase the gain before hitting neutral stability (i.e. making 1KG(js)) = 1 when LG(js)= 180°)

Phase margin - How many dequees we are from nortral stability (i.e. extremence in phase from 180° when IKGijw)=1

These are very aseful design concepts for the simple case that the system crosses 180° at exactly one frequency. Other cases require more complexity (and Nyquist stability analysis)



Considering a system in a feedback form:



We can see from the root locus that the system has poles on the imaginary axis when

1 KG(jw) 1=1 and LG(jw) = 180° or -180°

We can also understand the instability by looking at the closed loop transfer function near the crossover frequency of the open loop system. We is the frequency corresponding to a magnitude of I on the Bode plot:

| KG(jwe) | = 1 at crossover frequency we

Looking at the closed-loop transfer function from R to Y, which we will call T(3),

$$T(j\omega) = \frac{KG(j\omega)}{1 + KG(j\omega)}$$

When |KG(jw)|= 1 and LG(jw) = -180°, the denominator is not defined. This makes sense since any small imput will set up an oscillation at this neutral stability condition.

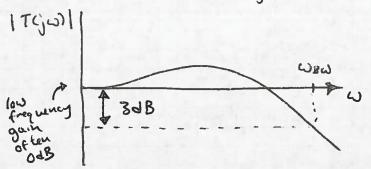
Most systems have the following behavior across their frequency range?

1K(jw) 1>>1 at low frequencies since we generally want 1T(jw) 1=1 for good reference tracking

(K(ju)) << 1 at high frequencies since we generally have more poles than zeros. This simply reflects the fact that physical systems have limits to how fast they can respond.

Because of this, ITijus | drops at higher frequencies, we call the bandwidth of the system the highest frequency it can reproduce sufficiently accurately,

Commonly, the bandwidth is defined to be the frequency of which the output of the system is down 30B) from its low frequency behavior.



-3dB is 0.707 so this
is the point when the
output sirusid is
70% of the reference
sirusoid's amplitude.

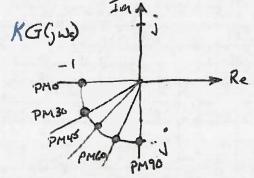
If performance is more critical for a particular system, the bandwidth can be defined as something other than a 3dB drop. If no other information is given, 3dB is generally being assumed. Control engineers also use the term "bandwith" more qualitatively when speaking about the frequency range of a system.

Bandwidth of a closed-loop system generally lies near the cossioner frequency of the open loop system.

Usually, we & WBW & ZWE

Bandwith is to the frequency domain what specifications like vise time and peak time are to the time domain. A higher bandwidth means the system responds well to faster reference commands

Just as the crossover frequency parallels response time, the phase margin parallels damping. To see this, we can look at the behavior of the closed-loop system at the crossover frequency and notice that it exhibits peaking. The magnitude of Ka(jwe) is always one (that is the definition of crossover) so the only thing that changes is the phase.



Kacjuel is just a complex number

A phase margin of a corresponds to a phase of 180°

A phase margin of 90° corresponds to a phase of -90°.

1 KG(jwe) 1 is always 1

So the magnitude of the closed- 100p system at the crossover frequency depends only upon the phase margin (since the magnitude is, by definition, 1).

We can therefore calculate the closed-loop transfer function's magnitude and phase at crossover for different values of the phase margin. Doing this, we can clearly see that phase margin is related to damping or resonance on the closed-loop Bode plot.

These are calculated by hand here as a reminder that these are simply complex numbers and the closed loop magnitude and phase follow directly from the open loop magnitude and phase at a given frequency.

PM 90° (phase = - 90°)

 $KG(j\omega_c) = -j$ $T(j\omega_c) = \frac{-j}{1-j} = \frac{-j}{1-j} \cdot \frac{1+j}{1+j} = -\frac{j+1}{2} = \frac{1}{2} - \frac{1}{2}j$ $|T(j\omega_c)| = \frac{\sqrt{2}}{2}$ or 0.707 (so amplitude is reduced) $\angle T(j\omega_c) = -45^\circ$

PM 600 (phase = -1200)

 $KG(j\omega_c) = -\frac{1}{2} - \frac{1}{2}j$ $T(j\omega_c) = \frac{1}{2} - \frac{1}{2}j = \frac{1}{2}j =$

1 TGWell = 1 L TGWe) = - 60°

I reproduce the output 181 relative to the reference but with a-60° degree phase 8h ft.

PM 450 (phase = -1350)

 $KG(j\omega_c) = \frac{\sqrt{2}}{2} - \frac{\sqrt{2}}{2}j$ $T(j\omega_c) = \frac{1}{2} - \frac{\sqrt{2}}{2(2-\sqrt{2})}j$

1T(jwe) 1 = 1.30 LT(jwe) = -67.50

Resonance as the closed loop system goes above lat we PM 30° (phase = -150°) $KG_2(j\omega_c) = \frac{-\sqrt{3}}{2} - \frac{1}{2}j$ $T(j\omega_c) = \frac{1}{2} - \frac{1}{2(2-\sqrt{3})}j$ $1T(j\omega_c) = \frac{1.93}{2}$ $LT(j\omega_c) = -75°$ Resonance amplitude increasing

As phase margin decreases further, the resonance peak inculases further...

PM10° => 1T(jwe) 1 = 5.74 LT(jwe) = -85° PM5° => 1T(jwe) 1 = 11.5 LT(jwe) = -87.5°

So this leads to thinking in the frequency domain about closed-loop boundwidth (or crossover frequency) and phase margin as analogous to our system response time and damping.



In the frequency doman, we often think of performance in terms of the bander-dff of the closed-loop system. This is closely related to the crossover frequency of the open loop system. Generally of

We & WBW & ZWE

We also want some amount of damping in the system which we can think of in terms of phase margin since of PM

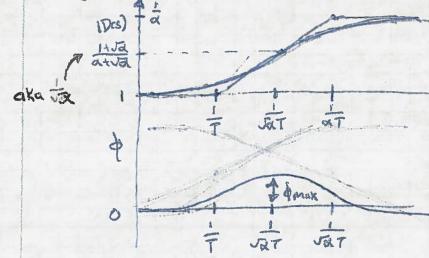
So instead of tuning natural frequency and damping vatio to get a desired time domain response I like vise time and overshoot, we tune the crossover frequency and phase margin to get a desired closed-loop bandwidth and an acceptable amount of resonance.

In terms of a design process, we can adjust the gave to get the desired crossover frequency then a did a lead compensator to get the desired phase margin.

The lead compensator has two parameters, Tandox and a form

D(s) = TS+1 OCOCCI

of is a number less than I that determines the separation of the zero and pole and T locates the zero in the frequency domain. The Bode plot



 $\phi = \tan^{-1}(T\omega) - \tan^{-1}(xT\omega)$ from the transfer function. The maximum phase occurs at $\sqrt{a}T$ and is $\sin \phi_{max} = \frac{1-\alpha}{1+\alpha}$ $\alpha = \frac{1-\sin \phi_{max}}{1+\sin \phi_{max}}$

If we like the crossover frequency we have with a given gain, we can add phase margin by choosing of and add that margin at the right I frequency by choosing T.

This will also have a change in the gain and crossover frequency unless we also change the gain of our compensator.

The lead compensator has a gam of divide by this value, we have a magnitude of 1 at tat and can add phase without changing the gain or crossover frequency.

So our final lead compensator looks like:

$$\mathcal{D}(s) = \left(\frac{\partial A J \partial A}{1 + J \partial A}\right) \left(\frac{T S + 1}{\partial A T S + 1}\right)$$

Or, even more simply, realizing that d+ Ja = Ja