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**HOMEWORK 2 SOLUTIONS**

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1. a. To avoid oscillations in the response, the roots of the characteristic equation must be real. This is because, if the roots are complex, then you will get terms like  $e^{-at} \cos(bt)$  or  $e^{-at} \sin(bt)$  in the response.

The transition from real to complex roots occurs when the discriminant is 0, i.e.

$$c = \sqrt{4mk} = \sqrt{4(50 \text{ kg})(5000 \text{ N/m})} = 1000 \text{ N/(m/s)}$$

Of course, larger values of  $c$  will also have no oscillations in the response, but this is the minimum value.

b.

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function hw2_1()

clear all;
close all;

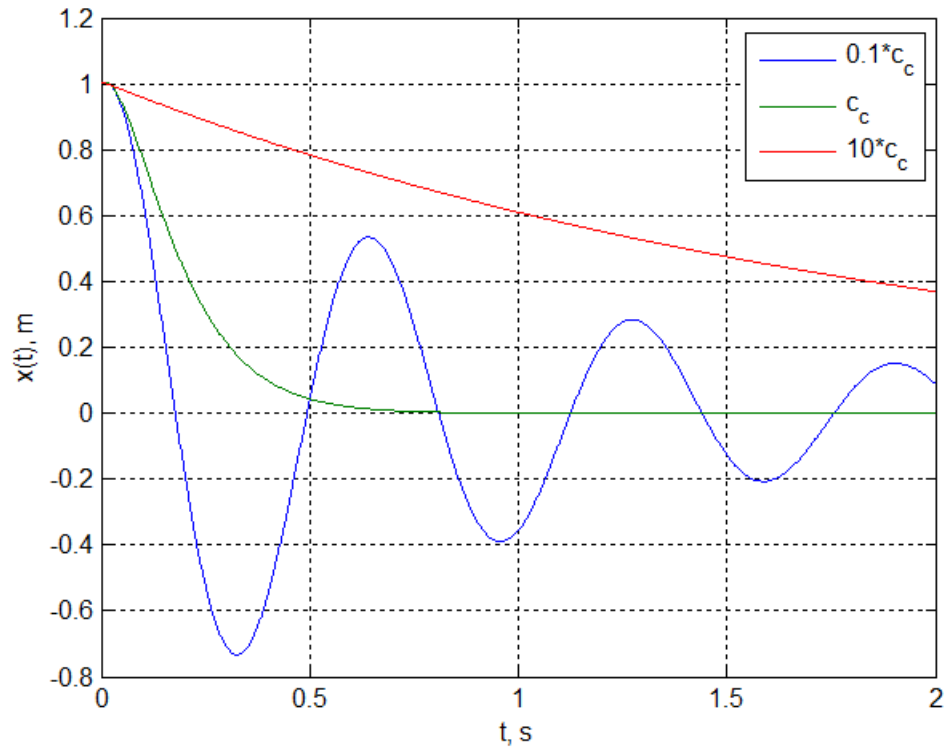
m = 50;
k = 5000;
c = sqrt(4*m*k);
c = [0.1*c c 10*c];

t = linspace(0,2,1001);
x = zeros(length(t),length(c));
x0 = [1; 1];
for i = 1:length(c)
    [dum,y] = ode45(@loc_eom_fun,t,x0);
    x(:,i) = y(:,1);
end

function xdot = loc_eom_fun(loc_t,loc_x)
    xdot = [loc_x(2); -c(i)/m*loc_x(2)-k/m*loc_x(1)];
end

plot(t,x);
ylabel('x(t), m');
xlabel('t, s');
legend('0.1*c_c','c_c','10*c_c');
grid on;

end
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2. Once the engine makes contact with the stopper, it becomes a simple spring-mass-damper system.

$$m\ddot{x} + c\dot{x} + kx = 0$$

$$m(s^2 X(s) - sx_0 - \dot{x}_0) + c(sX(s) - x_0) + kX(s) = 0$$

Rearranging and substituting in the numerical values gives

$$X(s) = \frac{m\dot{x}_0}{ms^2 + cs + k} = \frac{20,000}{2,000s^2 + 20,000s + 40,000} = \frac{10}{s^2 + 10s + 20}$$

Checking the discriminant of the denominator polynomial yields  $10^2 - 4 \cdot 20 = 20 > 0 \Rightarrow \text{real}$ , so the fraction can be expanded, using residue for example, as

$$X(s) = \frac{-2.2361}{s + 7.2361} + \frac{2.2361}{s + 2.7639}$$

whose inverse Laplace transform is  $x(t) = -2.2361e^{-7.2361t} + 2.2361e^{-2.7639t}$ .

a. To find the maximum compression, the velocity is set equal to 0:

$$\begin{aligned}\dot{x}(t) &= 16.1806e^{-7.2361t} - 6.18036e^{-2.7639t} = 0 \\ e^{-4.4722t} &= 0.3820 \\ t &= \frac{\ln 0.3820}{-4.4722} = 0.2125 \text{ s}\end{aligned}$$

b. The maximum compression is then

$$\begin{aligned}\max x(t) &= -2.2361e^{-7.2361 \cdot 0.2152} + 2.2361e^{-2.7639 \cdot 0.2152} \\ &= 0.762 \text{ m}\end{aligned}$$

3. We know that in general, the amplitude of motion decreases linearly when Coulomb friction is the sole damping force:

$$x_{i+1} = x_i - \frac{4\mu N}{k}$$

where, in this problem, the normal force is  $N = \frac{\sqrt{3}}{2}mg$ .

The motion stops when  $\dot{x} = 0$  and  $|x| \leq \frac{\mu N}{k}$ . In terms of half cycles  $n$ , this condition becomes

$$|x_0| - \frac{2\mu N}{k}n \leq \frac{\mu N}{k}$$

Solving for  $\mu$  gives

$$\mu \geq \frac{k|x_0|}{(2n+1)N} = \frac{2\omega_n^2|x_0|}{\sqrt{3}(2n+1)g} = 0.227$$

This gives us a lower bound for  $\mu$ . An upper bound for  $\mu$  can be found from the same equation assuming the mass stopped after 19 half cycles:

$$\mu \leq \frac{k|x_0|}{[2(n-1)+1]N} = \frac{2\omega_n^2|x_0|}{\sqrt{3}[2(n-1)+1]g} = 0.238$$

Therefore,  $0.227 \leq \mu \leq 0.238$ .

4.

$$s_{1,2} = \frac{-12 \pm \sqrt{144 - 8k}}{4} = -3 \pm \frac{1}{4}\sqrt{144 - 8k}$$

$0 < k < 18 \Rightarrow$  overdamped

$k = 18 \Rightarrow$  critically damped

$k > 18 \Rightarrow$  underdamped

