## **HOMEWORK 3 SOLUTIONS**

1. Since the collision is perfectly elastic, we can use conservation of momentum and energy to calculate the velocity of the ball and the block after the collision:

$$m_0 v_0(0-) = M v(0+) + m_0 v_0(0+)$$
 and  $\frac{1}{2} m_0 v_0^2(0-) = \frac{1}{2} M v^2(0+) + \frac{1}{2} m_0 v_0^2(0+)$ 

The velocity of the ball before impact is  $v_0(0-)=\sqrt{2gh}=6.26\,\mathrm{m/s}$  (positive downward). Solving this system of equations gives

$$v(0+) = \frac{2m_0v_0(0-)}{M+m_0} = 0.60 \text{ m/s} \text{ and } v_0(0+) = -\frac{(M-m_0)v_0(0-)}{M+m_0} = -5.66 \text{ m/s}$$

Since the ball is removed from the situation after the impact, we can now consider the spring-mass-damper system as a free vibration with initial position of 0 and initial velocity of 0.60 m/s.

The equation of motion is  $M\ddot{x} + c\dot{x} + kx = 0$ . Taking the Laplace transform yields

$$M(s^2X(s)-v(0+))+csX(s)+kX(s)=0$$

Solving for X(s) and plugging in the values gives

$$X(s) = \frac{Mv(0+)}{Ms^2 + cs + k} = \frac{1.2}{2s^2 + 5s + 100} = \frac{0.6}{s^2 + 2.5s + 50}$$

Checking the discriminant:  $2.5^2 - 4.50 = -193.75 < 0$ 

Completing the squares:  $s^2 + 2.5s + 50 = (s + 1.25)^2 + 48.4$ 

Hence, 
$$X(s) = \frac{0.6}{(s+1.25)^2 + 48.4} = \left(\frac{0.6}{7}\right) \frac{7}{(s+1.25)^2 + 48.4}$$

The inverse Laplace transform is then

$$x(t) = 0.086e^{-1.25t} \sin(7t)$$

2. The area of the piston is  $A = \frac{\pi}{4}d^2 = 0.0078$  m. Thus, the applied force on the piston is

$$f(t) = Ap(t) = 390(1 - e^{-3t})$$

The EOM is  $m\ddot{x} + kx = f(t)$ . Taking the Laplace transform and assuming 0 ICs gives  $ms^2X(s) + kX(s) = F(s)$ .

At this point, there are two methods for solving this problem:

## 1. Inverse Laplace transform method

Taking the Laplace transform of the input force gives

$$F(s) = \frac{390}{s} - \frac{390}{s+3}$$

Solving for X(s) and substituting in F(s) gives

$$X(s) = \frac{F(s)}{ms^2 + k} = \frac{390}{s(10s^2 + 1000)} - \frac{390}{(s+3)(10s^2 + 1000)}$$

Taking the inverse Laplace transform yields

$$x(t) = 0.39 - 0.36e^{-3t} - 0.11\sin(10t) - 0.032\cos(10t)$$

## 2. Convolution integral method

First, we must compute the transfer function, which is derived from the EOM assuming 0 ICs:

$$G(s) = \frac{X(s)}{F(s)} = \frac{1}{ms^2 + k} = \frac{1}{10s^2 + 1000}$$

This inverse Laplace transform of G(s) is the unit impulse response:

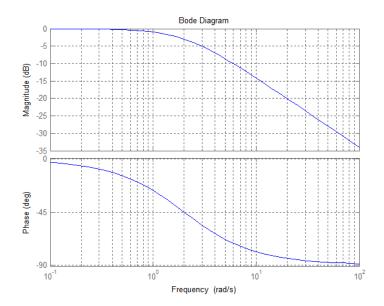
$$g(t) = 0.01\sin(10t)$$

Now, we can use the convolution integral to get x(t) as follows:

$$x(t) = \int_0^t f(\tau)g(t-\tau)d\tau = \int_0^t 390(1-e^{-3\tau})0.01\sin[10(t-\tau)]d\tau$$
$$= 0.39 - 0.36e^{-3t} - 0.11\sin(10t) - 0.032\cos(10t)$$

3. a. The Bode plot of  $\frac{1}{0.5s+1}$  has a corner frequency at  $\omega = 2 \text{ rad/s}$ , a slope of 0 dB/decade to the left of this and a slope of -20 dB/decade to the right. The gain as  $\omega \to 0$  is 0 dB. The phase varies from 0 deg. to -90 deg.

Hence, the Bode plot looks like

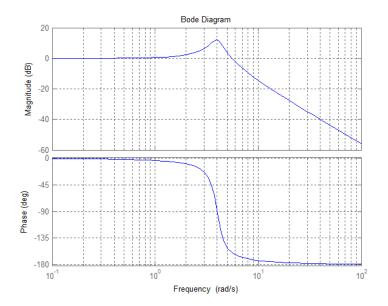


b. The term  $\frac{16}{s^2 + s + 16}$  has complex conjugate roots and so cannot be factored into first-order

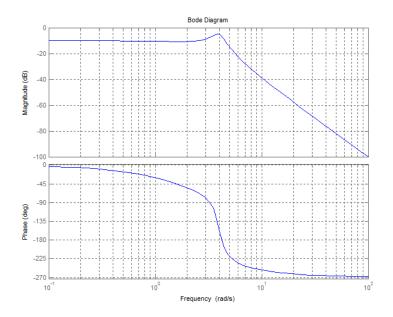
terms. Writing the denominator as  $s^2 + 2\zeta\omega_n s + \omega_n^2$ , we can see that  $\omega_n = 4$  rad/s, which is the corner frequency for this term, and  $\zeta = 0.125$ , which indicates that there will be a peak near the corner frequency. Since it's a second-order term, it will have a slope of 0 dB/decade to the left of this and a slope of -40 dB/decade to the right. The gain as  $\omega \to 0$  is 0 dB. The phase varies from 0 deg. to -180 deg.

The magnitude of the peak is given by 
$$M_p = \frac{1}{2\zeta\sqrt{1-\zeta^2}} = 4.03 = 12.1 \,\text{dB}$$
.

Hence, the Bode plot looks like



- c. The term  $\frac{5}{16}$  is a straight line with magnitude  $20\log\left(\frac{5}{16}\right) = -10.1 \, dB$  and 0 phase angle.
- d. Combining these terms yields



4. a. 
$$G(s) = \frac{1}{0.1s+1}$$

b. 
$$G(s) = \frac{100s + 1}{0.1s + 1}$$
  
d.  $G(s) = \frac{1}{(s+1)^2}$ 

c. 
$$G(s) = 10(10s + 1)$$

d. 
$$G(s) = \frac{1}{(s+1)^2}$$