

PROBLEM 2.11 RELATIVE TO AN XYZ COORDINATE SYSTEM, THE STATE OF STRESS AT A POINT IS KNOWN TO BE

$$[\sigma] = \begin{bmatrix} -10 & 20 & 30 \\ 20 & 10 & -20 \\ 30 & -20 & 40 \end{bmatrix} \text{ MPa}$$

(1)

- (a) EVALUATE THE NORMAL AND SHEAR STRESSES ON A SURFACE AT THE POINT WHERE THE SURFACE IS GIVEN BY THE THREE POINTS $(1, 0, 0)$, $(0, 2, 0)$, $(0, 0, -1)$
- (b) DETERMINE THE DIRECTIONAL COSINES FOR THE SHEAR STRESS FOUND IN PART (a) AND IN A ROUGH SKETCH SHOW THE DIRECTION OF THE NORMAL AND SHEAR STRESSES.

GIVEN:

CONSTRAINTS

1. THE STATE OF STRESS AT A POINT IN A STRUCTURE IS DEFINED BY (1)
2. THE PLANE OF INTEREST IS DEFINED BY $(1, 0, 0)$, $(0, 2, 0)$, $(0, 0, -1)$

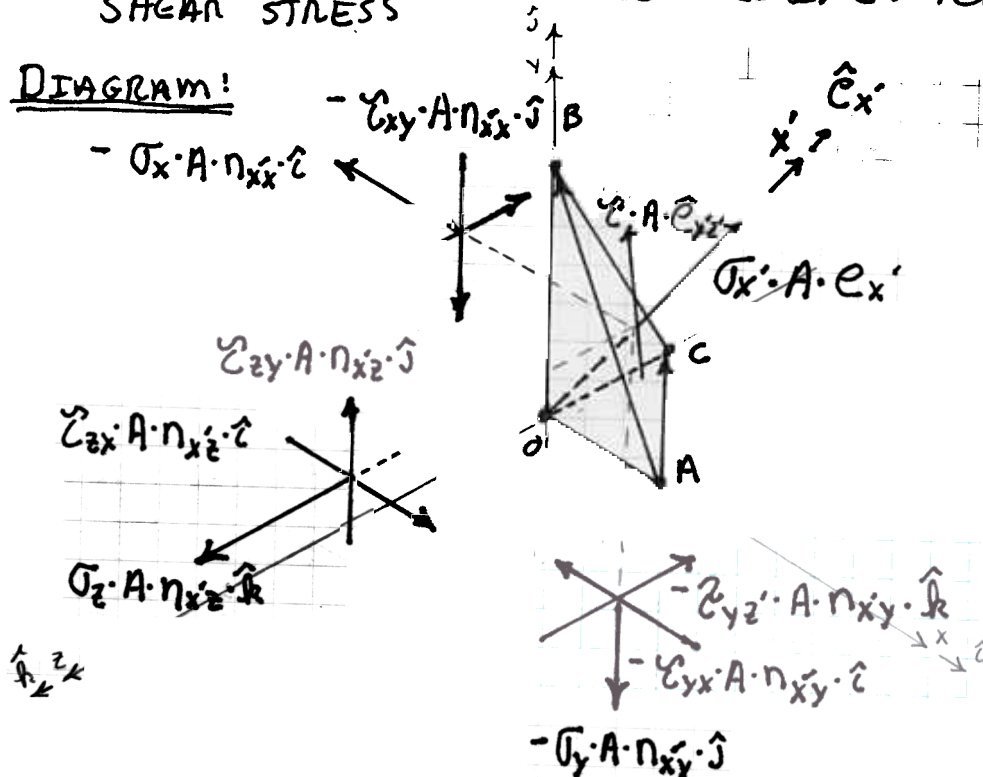
ASSUMPTIONS

1. THE STRESS IS AT A SINGLE POINT.
2. THE STRESS TENSOR IS CARTESIAN.

FIND:

1. CALCULATE THE NORMAL AND SHEAR STRESS ON THE SURFACE
2. DETERMINE THE DIRECTIONAL COSINES FOR THE NORMAL AND SHEAR STRESS

DIAGRAM:



$$A_x = n_{x'} \cdot A$$

$$A_y = n_{y'} \cdot A$$

$$A_z = n_{z'} \cdot A$$

(A)

SOLUTION:

TO FIND THE NORMAL TO THE SURFACE "ABC", THE THREE POINTS

$$A: (1,0,0); B(0,2,0); C(0,0,-1)$$

ARE USED TO FORM TWO POSITION VECTORS

$$\vec{r}_{AC} = (0-1)\hat{i} + (0-0)\hat{j} + (1-0)\hat{k} = -\hat{i} - \hat{k} \quad (2)$$

$$\vec{r}_{AB} = (0-1)\hat{i} + (2-0)\hat{j} + (0-0)\hat{k} = -\hat{i} + 2\hat{j} \quad (3)$$

TAKING THE CROSS PRODUCT OF (2) AND (3) YIELDS A VECTOR PERPENDICULAR TO THE SURFACE NORMAL

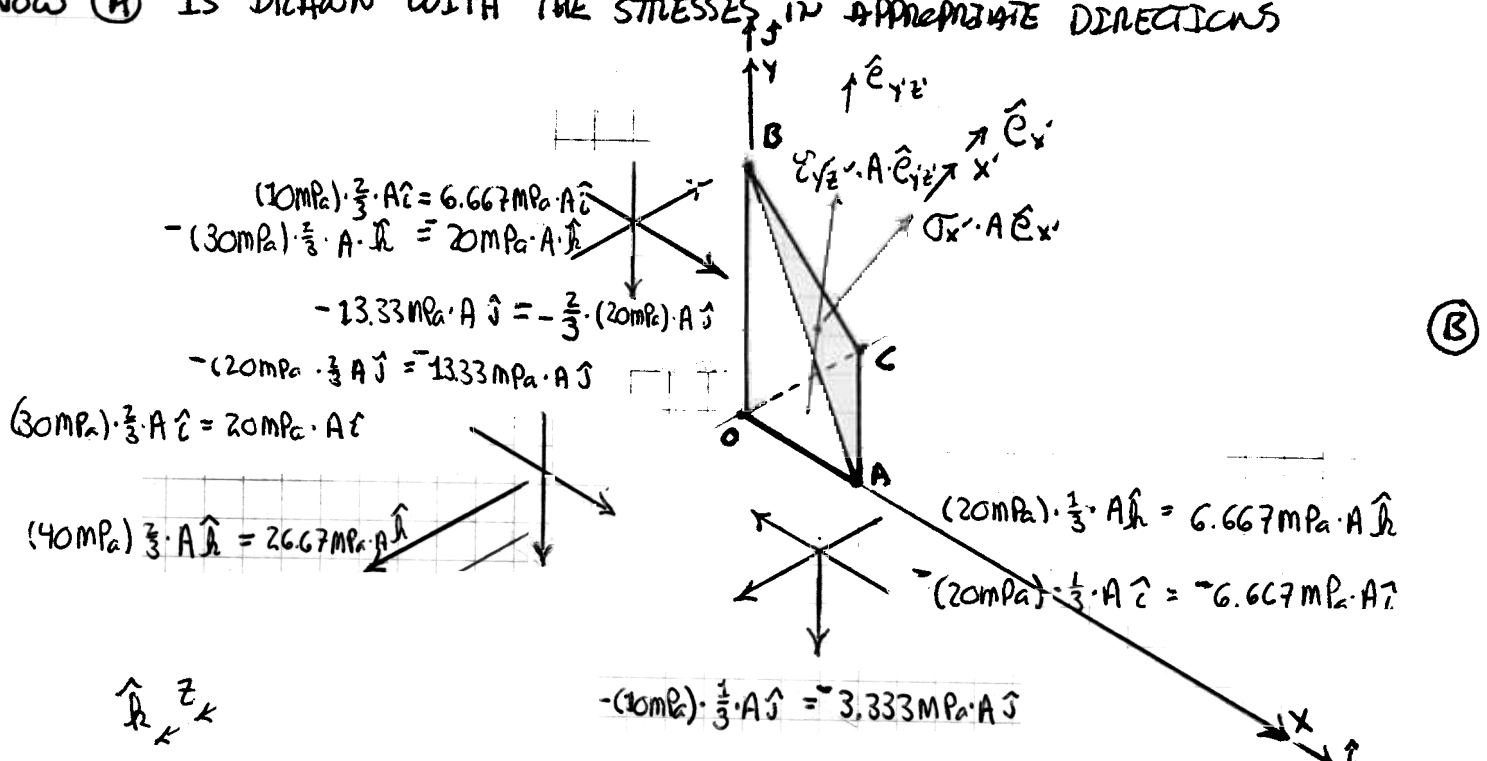
$$\vec{r}_{x'} = \vec{r}_{AC} \times \vec{r}_{AB} = \begin{vmatrix} \hat{i} & \hat{j} & \hat{k} \\ -1 & 0 & -1 \\ -1 & 2 & 0 \end{vmatrix} = 2\hat{i} + \hat{j} - 2\hat{k}$$

$$= \sqrt{9} \left(\frac{2}{\sqrt{9}}\hat{i} - \frac{1}{\sqrt{9}}\hat{j} - \frac{2}{\sqrt{9}}\hat{k} \right) = 3 \left(\frac{2}{3}\hat{i} - \frac{1}{3}\hat{j} - \frac{2}{3}\hat{k} \right) = r_{x'} \cdot \hat{e}_{x'}$$

THE UNIT VECTOR $\hat{e}_{x'}$ DEFINES THE DIRECTION OF THE SURFACE NORMAL

$$\hat{e}_{x'} = \frac{2}{3}\hat{i} + \frac{1}{3}\hat{j} - \frac{2}{3}\hat{k} = n_{x'x}\hat{i} + n_{x'y}\hat{j} - n_{x'z}\hat{k} \quad (4)$$

NOW (A) IS DRAWN WITH THE STRESSES IN APPROPRIATE DIRECTIONS



CONSIDERING THE EQUILIBRIUM OF ELEMENT "ABCO"

$$\begin{aligned}\Sigma \vec{F} = \vec{0} &= \sigma_x' \cdot A \cdot \hat{e}_x' + \tau_{xy'} \cdot A \cdot \hat{e}_{y'z'} + (6.667 \text{ MPa} + 20 \text{ MPa} - 6.667 \text{ MPa}) \cdot A \cdot \hat{i} \\ &+ A \cdot (-13.33 \text{ MPa} - 13.33 \text{ MPa} - 3.333 \text{ MPa}) \hat{j} + (-20 \text{ MPa} + 26.67 \text{ MPa} + 6.67 \text{ MPa}) A \cdot \hat{k} \\ &= \sigma_x' \cdot \hat{e}_x' + \tau_{y'z'} \cdot \hat{e}_{y'z'} + 20 \text{ MPa} \hat{i} - 30 \text{ MPa} \hat{j} + 13.34 \text{ MPa} \hat{k}\end{aligned}$$

$$\sigma_x' \hat{e}_x' + \tau_{y'z'} \hat{e}_{y'z'} = -20 \text{ MPa} \hat{i} + 30 \text{ MPa} \hat{j} - 13.34 \text{ MPa} \hat{k} \quad (5)$$

DOTTING (5) WITH THE UNIT VECTOR NORMAL TO THE SURFACE \hat{e}_x' , (4)

$$(\sigma_x' \hat{e}_x' + \tau_{y'z'} \hat{e}_{y'z'}) \cdot \hat{e}_x' = (-20 \text{ MPa} \hat{i} + 30 \text{ MPa} \hat{j} - 13.34 \text{ MPa} \hat{k}) \cdot \left(\frac{2}{3} \hat{i} + \frac{1}{3} \hat{j} - \frac{2}{3} \hat{k}\right)$$

$$\underbrace{\sigma_x' \hat{e}_x' \cdot \hat{e}_x'}_1 + \underbrace{\tau_{y'z'} \hat{e}_{y'z'} \cdot \hat{e}_x'}_0 = (-20 \text{ MPa})\left(\frac{2}{3}\right) + (30 \text{ MPa})\left(\frac{1}{3}\right) + (-13.34 \text{ MPa})\left(-\frac{2}{3}\right)$$

$$\sigma_x' = 5.556 \text{ MPa} = \boxed{5.56 \text{ MPa}}$$

NOW (5) CAN BE SOLVED FOR $\tau_{y'z'} \hat{e}_{y'z'}$

$$\begin{aligned}\tau_{y'z'} \hat{e}_{y'z'} &= (-20 \text{ MPa} \hat{i} + 30 \text{ MPa} \hat{j} - 13.34 \text{ MPa} \hat{k}) - \sigma_x' \hat{e}_x' \\ &= (-20 \text{ MPa} \hat{i} + 30 \text{ MPa} \hat{j} - 13.34 \text{ MPa} \hat{k}) - 5.556 \text{ MPa} \left(\frac{2}{3} \hat{i} + \frac{1}{3} \hat{j} - \frac{2}{3} \hat{k}\right)\end{aligned}$$

$$= (-20 \text{ MPa} \hat{i} + 30 \text{ MPa} \hat{j} - 13.34 \text{ MPa} \hat{k}) - 3.704 \text{ MPa} \hat{i} - 1.852 \text{ MPa} \hat{j} + 3.704 \text{ MPa} \hat{k}$$

$$= -23.70 \text{ MPa} \hat{i} + 28.15 \text{ MPa} \hat{j} - 9.636 \text{ MPa} \hat{k}$$

$$= 38.04 \text{ MPa} \cdot (-.6230 \hat{i} + .7400 \hat{j} - 2.533 \hat{k})$$

$$\tau_{y'z'} = \boxed{38.04 \text{ MPa}}$$

$$\hat{e}_{y'z'} = \boxed{-.6230 \hat{i} + .7400 \hat{j} - 2.533 \hat{k}}$$

SUMMARY:

THE SOLUTION IS OBTAINED THROUGH A CORRECT APPLICATION OF EQUILIBRIUM OF THE FORCES THAT RESULT FROM INTEGRATING THE STRESSES OVER THE SURFACES THAT THEY ACT UPON.