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Signature: _____

Print Name: SOLUTION

Exam Date: 19 FEBRUARY 2016

Diagram illustrating a bicycle frame mechanism with four segments (1, 2, 3, 4) and joints (I12, I13, I14, I23, I24, I34). The input force F_{in} is applied at joint I34, and the output force F_{out} is applied at joint I12. Dimensions include $d_{out} = 0.7$, $d_{in} \approx 3.7$, $r_{out} = 8$, and $r_{in} \approx 2.8$. Axes I_1I_2 , I_2I_3 , I_3I_4 , and I_4I_1 are shown in cyan.

1a. Determine the Mechanical Advantage of this system. Use the figure above for the construction of the relevant features that will allow you to calculate the mechanical advantage. Write all calculations below.

$$MA = \frac{d_{in}}{d_{out}} \cdot \frac{F_{out}}{F_{in}} = \frac{3.7}{0.7} \cdot \frac{0.8}{2.8} = \boxed{1.5}$$

1b. If the input force F_{in} is 200lb, what force will the shock see?

$$MA = \frac{F_{out}}{F_{in}}$$

$$1.5 = \frac{F_{out}}{F_{in}} = \frac{F_{out}}{200lb} \Rightarrow F_{out} = 1.5(200lb) = \boxed{300lb}$$

PROBLEM 2: The mechanism below has an input link (link 2) length of 1.6 inches. The input angular velocity ω_2 is 20-1/s.

$$V_{I_{23}} = \omega_2 \cdot r_2 = (20 \text{ 1/s}) (1.6 \text{ in}) = 32 \text{ in/s}$$

2a: Locate ALL instant centers associated with this mechanism:

2b: Determine the Linear velocity of Link 6.

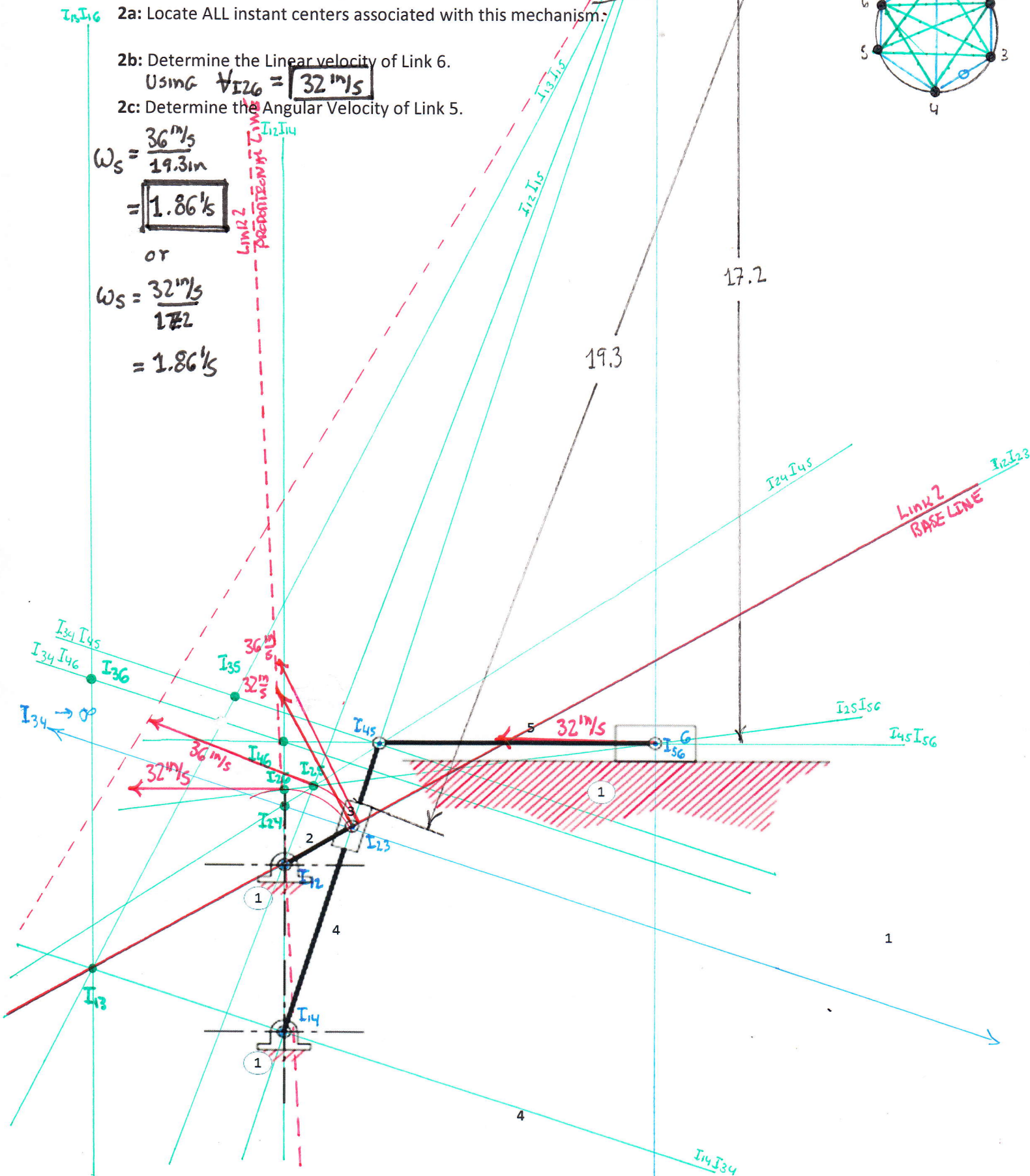
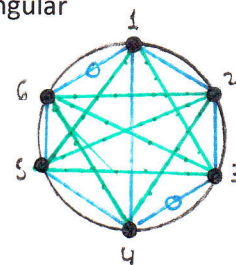
Using $V_{I_{26}} = \boxed{32 \text{ in/s}}$

2c: Determine the Angular Velocity of Link 5.

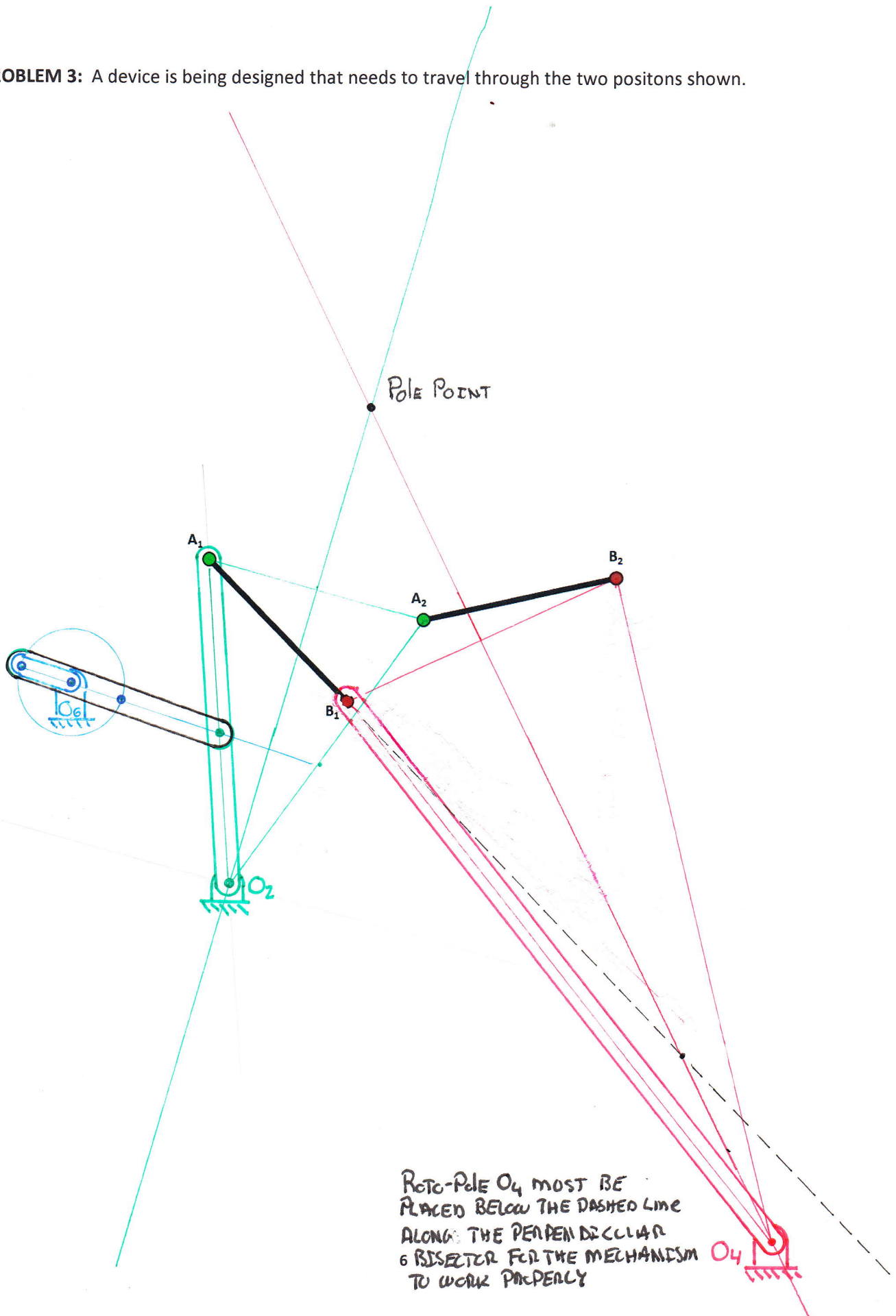
$$\omega_5 = \frac{36 \text{ in/s}}{19.3 \text{ in}} = \boxed{1.86 \text{ 1/s}}$$

or

$$\omega_5 = \frac{32 \text{ in/s}}{17.2 \text{ in}} = 1.86 \text{ 1/s}$$



PROBLEM 3: A device is being designed that needs to travel through the two positons shown.



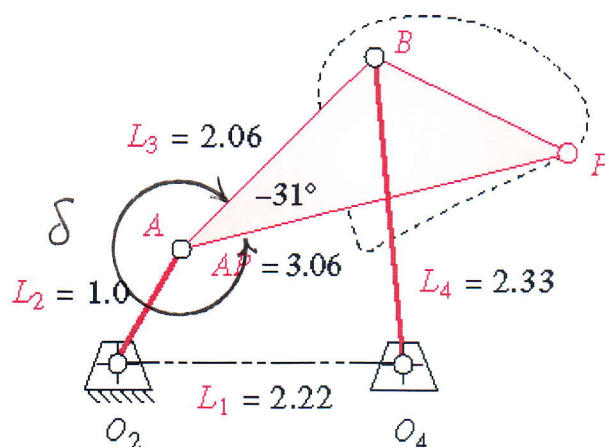
3a: Can the mechanism be designed for rocker output if all the ground connections (roto-poles) must be below the two positions shown? Explain your answer in terms of a construction on the figure below.

NO, BECAUSE THE POLE-POINT IS ABOVE THE MECHANISM AND THIS IS THE ONLY LOCATION OF THE ROO-POLE THAT WILL GIVE ROCKER MOTION FOR THE POSITIONS SHOWN.

3b: Use the figure above to design the rest of the mechanism that will move from position 1 above to position 2 and back in a single cycle. Design the drive dyad (NON-quick return) for the mechanism. The link that the drive dyad attaches to must rotate through 40° in traveling from position 1 to position 2. The complete drive mechanism must be to the left of the mechanism and completely on the page.

PROBLEM 4: The drive link of the mechanism below is rotating with an angular velocity of 5 1/s and an angular acceleration of -20 1/s^2 when the drive link makes a 70° angle with the horizontal. Input the appropriate parameters into the program that you wrote for this class to determine all the angular and linear positions, velocities, and accelerations for this mechanics including point P. (PRINT OUT THE RESULTS OF YOUR PROGRAM AND STAPLE IT AFTER THIS PAGE)

$$\delta = 360 - 31 = \underline{\underline{329^\circ}}$$



$p = 3.06$
 $\delta = 329$

$p = 3.06$
 $\delta = 329$



				e _r			e _h		
	x comp	y comp	mag	angle	i	j	i	j	
r04=	2.22	0.00	2.220	0.0	1.000	0.000	0.000	1.000	
rA=	0.34	0.94	1.000	70.0	0.342	0.940	-0.940	0.342	
rB=	1.54	1.37	2.060	41.5	0.749	0.663	-0.663	0.749	
rB04=		2.31	2.330	98.3	-0.144	0.990	-0.990	-0.144	
rB=	1.88	2.31	2.978	50.7	0.633	0.774	-0.774	0.633	
rPA=	3.01	0.56	3.060	10.5	0.983	0.183	-0.183	0.983	
rP=	3.35	1.50	3.671	24.1	0.913	0.408	-0.408	0.913	
vA=	-4.70	1.71	5.000	160.0	-0.940	0.342	-0.342	-0.940	
vB=	1.88	-2.12	2.893	-48.5	0.663	-0.749	0.749	0.663	
vB=	-2.82	-0.41	2.849	-171.7	-0.990	-0.144	0.144	-0.990	
vPA=	0.77	-4.14	4.209	-79.5	0.183	-0.983	0.983	0.183	
vP=	-3.93	-2.43	4.619	-148.3	-0.851	-0.526	0.526	-0.851	
aA=	10.24	-30.33	32.016	-71.3	0.320	-0.947	0.947	0.320	
aBA	-26.82	24.40	36.254	137.7	-0.740	0.673	-0.673	-0.740	
aB	-16.57	-5.94	17.605	-160.3	-0.941	-0.337	0.337	-0.941	
aPA=	-15.48	51.58	53.853	106.7	-0.287	0.958	-0.958	-0.287	
aP=	-5.24	21.25	21.883	103.8	-0.239	0.971	-0.971	-0.239	
ALT	x comp	y comp	mag	angle	i	j	i	j	
r04=	2.22	0.00	2.220	0.0	1.000	0.000	0.000	1.000	
rA=	0.34	0.94	1.000	70.0	0.342	0.940	-0.940	0.342	
rB=	-0.17	-2.05	2.060	-94.7	-0.082	-0.997	0.997	-0.082	
rB04=	-2.05	-1.11	2.330	-151.5	-0.878	-0.478	0.478	-0.878	
rB=	0.17	-1.11	1.127	-81.2	0.164	-0.988	0.988	0.164	
rPA=	-1.79	-2.48	3.060	-125.7	-0.584	-0.812	0.812	-0.584	
rP=	-1.44	-1.55	2.115	-133.1	-0.683	-0.731	0.731	-0.683	
vA=	-4.70	1.71	5.000	160.0	-0.940	0.342	-0.342	-0.940	
vB=	3.94	-0.32	3.958	-4.7	0.997	-0.082	0.082	0.997	
vB=	-0.75	1.39	1.578	118.5	-0.478	0.878	-0.878	-0.478	
vPA=	4.77	-3.43	5.879	-35.7	0.812	-0.584	0.584	0.812	
vP=	0.08	-1.72	1.722	-87.5	0.044	-0.999	0.999	0.044	
aA=	10.24	-30.33	32.016	-71.3	0.320	-0.947	0.947	0.320	
aBA	3.48	7.34	8.126	64.7	0.428	0.904	-0.904	0.428	
aB	13.72	-22.99	26.772	-59.2	0.513	-0.859	0.859	0.513	
aPA=	10.05	6.69	12.071	33.7	0.832	0.554	-0.554	0.832	
aP=	20.29	-23.64	31.155	-49.4	0.651	-0.759	0.759	0.651	

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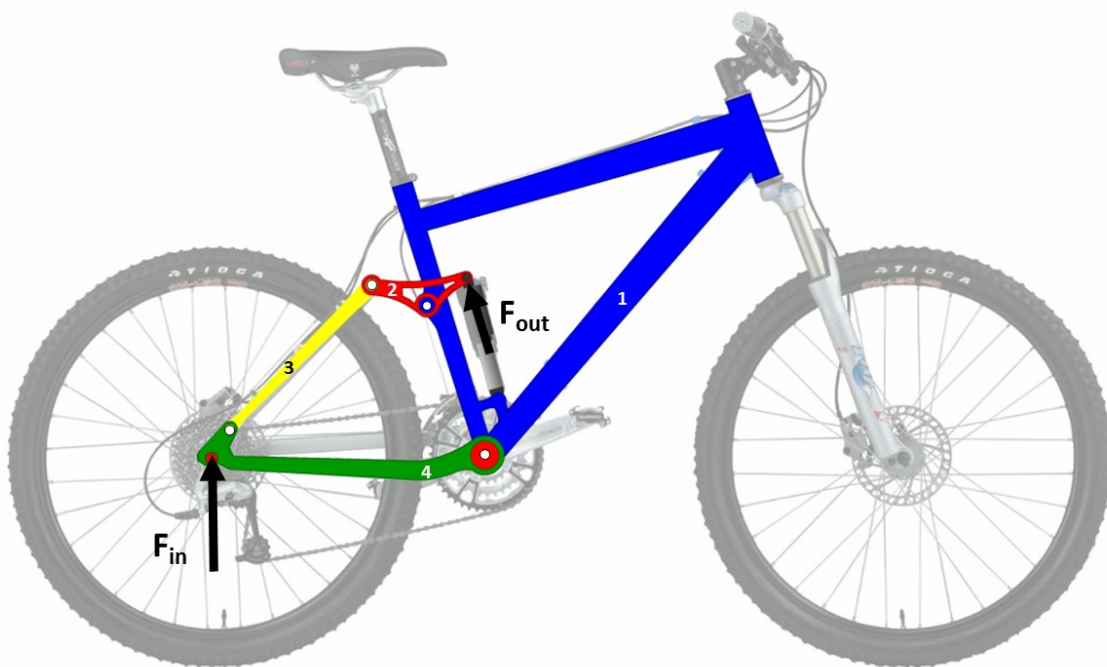
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PROBLEM 1: A shock is being designed for the bike below. The frame can be considered the ground link and the other links are numbered. The direction and location of the input force F_{in} is shown on the drawing along with the location and direction of the output Force F_{out} .



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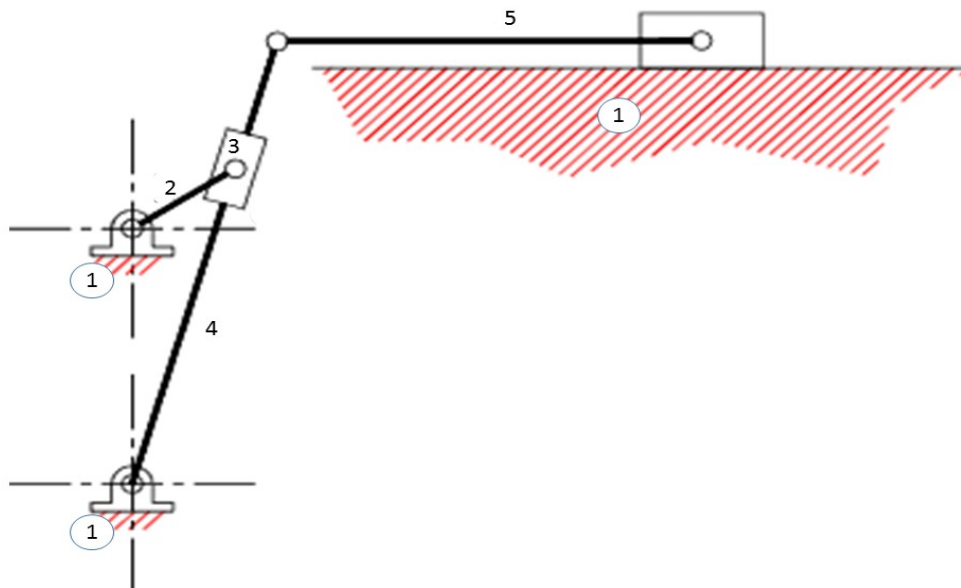
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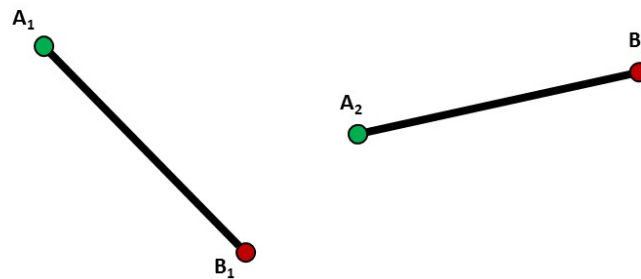
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