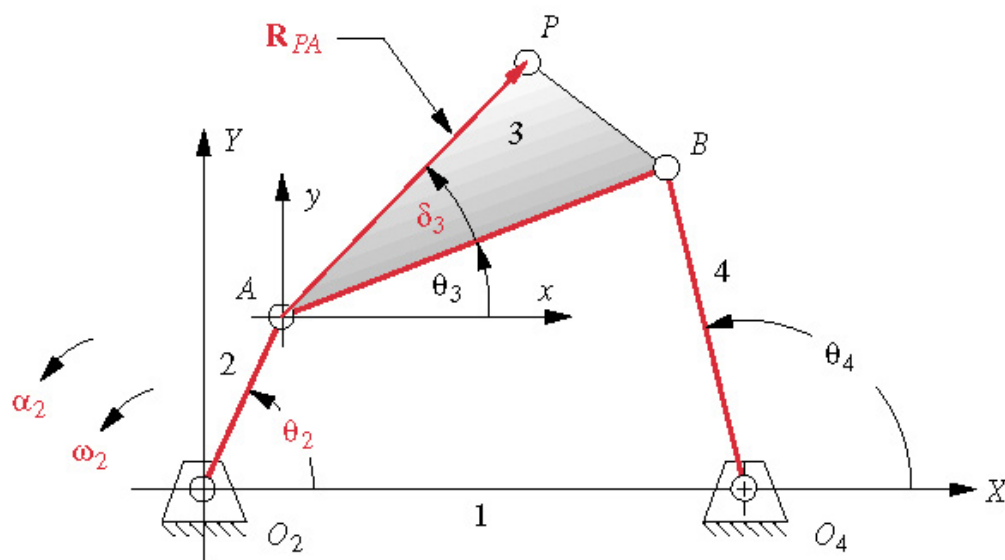


NAME: SOLUTION

PROBLEM 1 (30 pts): A four bar linkage has the following dimensions. All lengths are in inches and all angles are in degrees.

Link 1	Link 2	Link 3	Link 4	θ_2	R_{PA}	δ_3
4	6	10	7	88	10	330

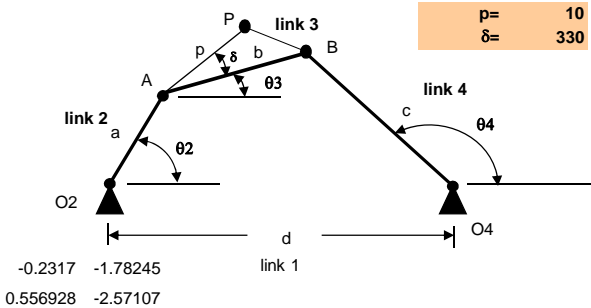
Link 2 is the drive link and it is known to be rotating at $\omega_2 = -80 \frac{1}{s}$ and $\alpha_2 = 30 \frac{1}{s^2}$. Using the program that you have been developing, calculate all the critical parameters associated with this linkage in both of the possible configurations. Print out the results of your program and staple it directly behind this page.



4-Bar Linkage

a=	6	Link 2
b=	10	Link 3
c=	7	Link 4
d=	4	Link 1
$\theta_2 =$	88	1.535889742
$\dot{\theta}_2 =$	-80	$\frac{1}{s}$
$\ddot{\theta}_2 =$	30	$\frac{1}{s^2}$
By=	3.70	-3.78
Bx=	9.94	-1.89
$\theta_3 =$	-13.3	-102.1
$\theta_4 =$	31.9	-147.3
$\dot{\theta}_3 =$	-5.6156E+01	-5.5638E+01
$\dot{\theta}_4 =$	-9.4798E+01	-1.6996E+01
$\ddot{\theta}_3 =$	-2.6936E+03	3.1098E+02
$\ddot{\theta}_4 =$	-4.0547E+03	1.6721E+03

K1= 4.0891E+00
K2= 1.5819E+00
K3= -8.0449E-02
K4= -1.3988E+01



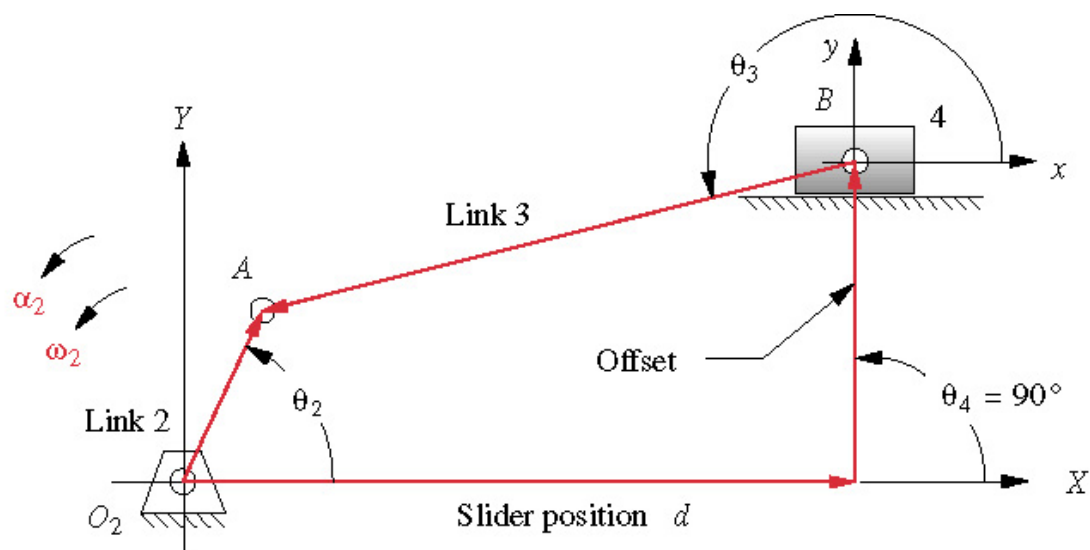
p= 10
delta= 330

	x comp	y comp	mag	angle	i	j
rO4=	4.00	0.00	4.000	0.0	1.000	0.000
rA=	0.21	6.00	6.000	88.0	0.035	0.999
rBA=	9.73	-2.30	10.000	-13.3	0.973	-0.230
rBO4=	5.94	3.70	7.000	31.9	0.849	0.529
rB=	9.94	3.70	10.608	20.4	0.937	0.349
rPA=	7.28	-6.86	10.000	-43.3	0.728	-0.686
rP=	7.49	-0.86	7.539	-6.5	0.993	-0.114
vA=	479.71	-16.75	480.000	-2.0	0.999	-0.035
vBA=	-128.95	-546.55	561.559	-103.3	-0.230	-0.973
vB=	350.76	-563.31	663.584	-58.1	0.529	-0.849
vPA=	-384.95	-408.85	561.559	-133.3	-0.686	-0.728
vP=	94.76	-425.61	436.027	-77.4	0.217	-0.976
aA=	-1520.03	-38370.33	38400.422	-92.3	-0.040	-0.999
aBA=	-36877.40	-18974.51	41472.574	-152.8	-0.889	-0.458
aB=	-38397.43	-57344.83	69012.986	-123.8	-0.556	-0.831
aPA=	-41424.02	2006.30	41472.574	177.2	-0.999	0.048
aP=	-42944.05	-36364.03	56271.964	-139.7	-0.763	-0.646
ALT	x comp	y comp	mag	angle	i	j
rO4=	4.00	0.00	4.000	0.0	1.000	0.000
rA=	0.21	6.00	6.000	88.0	0.035	0.999
rBA=	-2.10	-9.78	10.000	-102.1	-0.210	-0.978
rBO4=	-5.89	-3.78	7.000	-147.3	-0.842	-0.540
rB=	-1.89	-3.78	4.227	-116.6	-0.447	-0.894
rPA=	-6.71	-7.42	10.000	-132.1	-0.671	-0.742
rP=	-6.50	-1.42	6.652	-167.7	-0.977	-0.214
vA=	479.71	-16.75	480.000	-2.0	0.999	-0.035
vBA=	-543.96	116.88	556.375	167.9	-0.978	0.210
vB=	-64.25	100.13	118.970	122.7	-0.540	0.842
vPA=	-412.64	373.20	556.375	137.9	-0.742	0.671
vP=	67.06	356.45	362.702	79.3	0.185	0.983
aA=	-1520.03	-38370.33	38400.422	-92.3	FALSE	-0.999
aBA=	9543.26	29611.30	31111.140	72.1	0.307	0.952
aB=	8023.23	-8759.02	11878.243	-47.5	0.675	-0.737
aPA=	23070.36	20872.51	31111.140	42.1	0.742	0.671
aP=	21550.32	-17497.81	27759.503	-39.1	0.776	-0.630

PROBLEM 2 (30 pts): A slider crank linkage has the following dimensions. All lengths are in inches and all angles are in degrees.

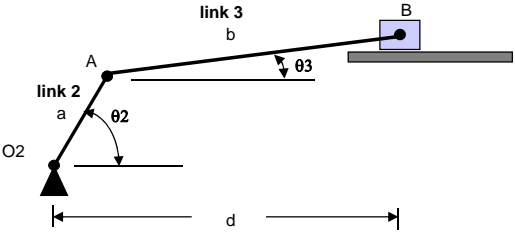
Link 2	Link 3	Offset	θ_2
5	20	-5	225

Link 2 is the drive link and it is known to be rotating at $\omega_2 = -50 \frac{1}{s}$ and $\alpha_2 = 10 \frac{1}{s^2}$. Using the program that you have been developing, calculate all the critical parameters associated with this linkage in both of the possible configurations. Print out the results of your program and staple it directly behind this page.



Slider Crank

a=	5	Link 2
b=	20	Link 3
c=	-5	Link 1
$\theta_2 =$	225	3.926990817
$\dot{\theta}_2 =$	-50	$\frac{1}{s}$
$\ddot{\theta}_2 =$	10	$\frac{1}{s^2}$
B _y =	-5.00	-5.00
B _x =	16.41	-23.48
$\theta_3 =$	-4.2	-175.8
$\dot{\theta}_3 =$	-8.86	8.86
$\ddot{\theta}_3 =$	-447.13	447.13
v _B =	-189.76	-163.80
a _B =	6652.68	11095.70

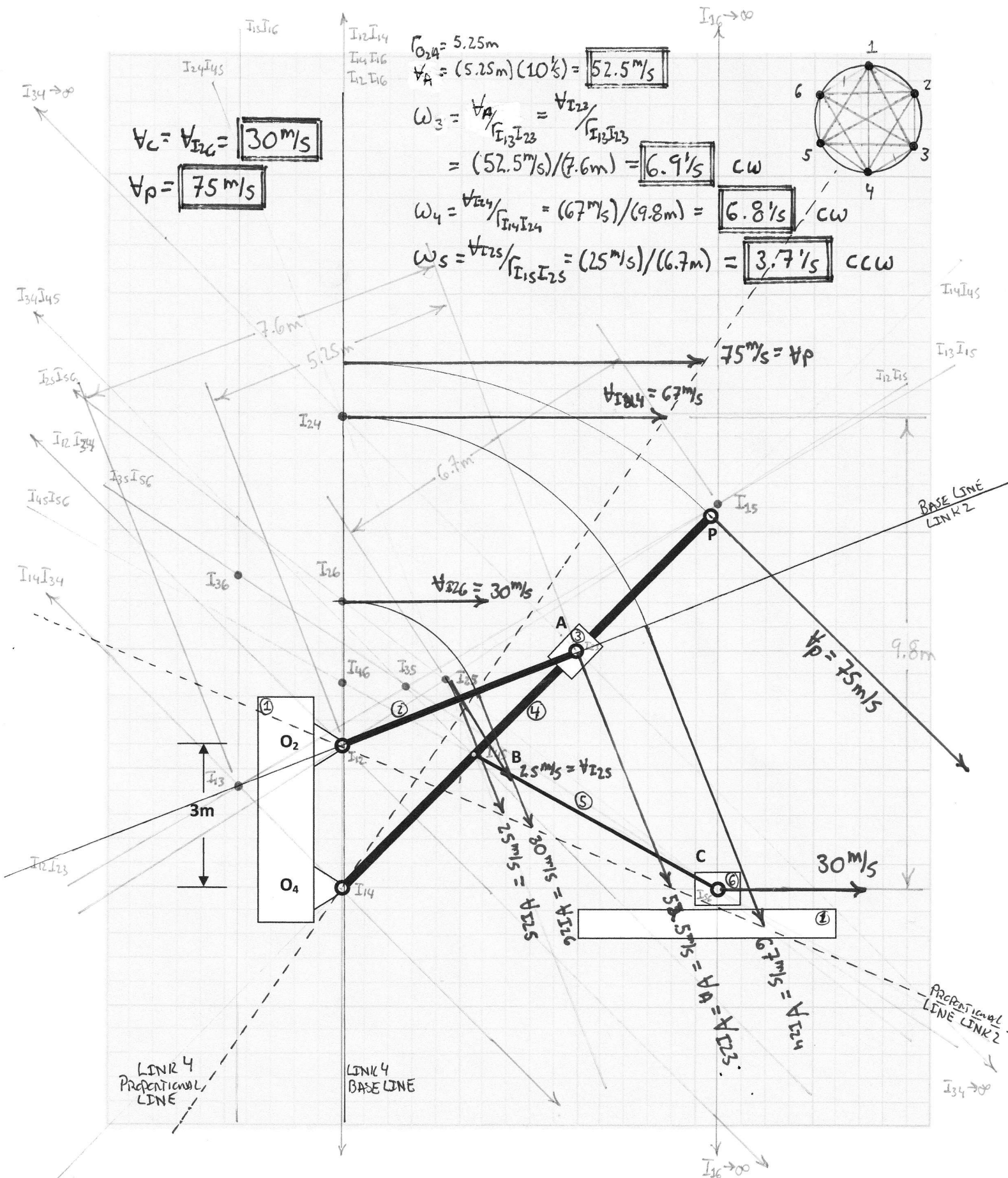


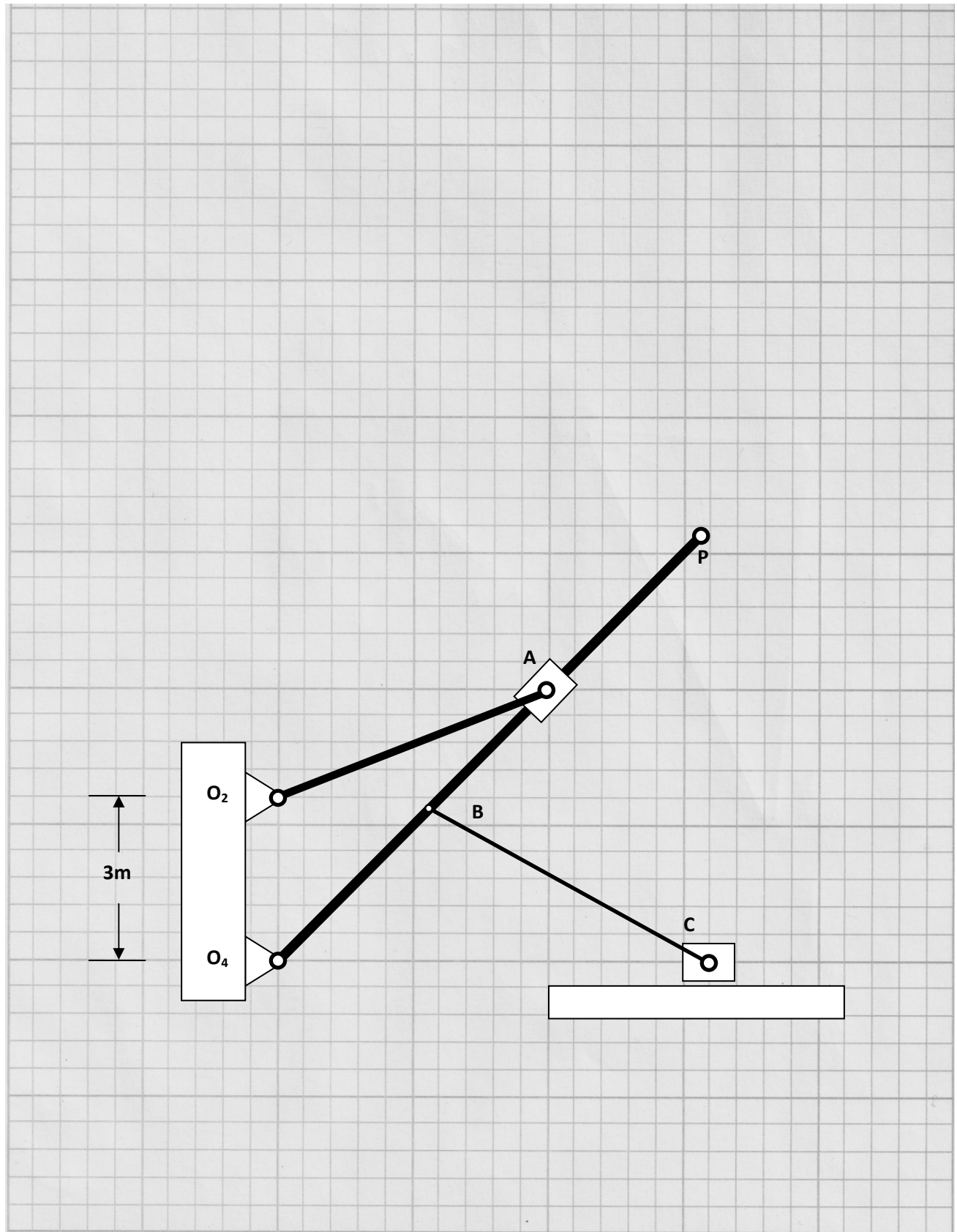
	x comp	y comp	mag	angle	i	j
r _B =	16.41	-5.00	17.16	-16.9	0.957	-0.291
r _A =	-3.54	-3.54	5.00	-135.0	-0.707	-0.707
r _{BA} =	19.95	-1.46	20.00	-4.2	0.997	-0.073
v _B =	-189.76	0.00	189.76	180.0	-1.000	0.000
v _A =	-176.78	176.78	250.00	135.0	-0.707	0.707
v _{BA} =	-12.98	-176.78	177.25	-94.2	-0.073	-0.997
a _B =	6652.68	0.00	6652.68	0.0	1.000	0.000
a _A =	8874.19	8803.48	12500.10	44.8	0.710	0.704
a _{BA} =	-2221.51	-8803.48	9079.45	-104.2	-0.245	-0.970
alt	x comp	y comp	mag	angle	i	j
r _B =	-23.48	-5.00	24.01	-168.0	-0.978	-0.208
r _A =	-3.54	-3.54	5.00	-135.0	-0.707	-0.707
r _{BA} =	-19.95	-1.46	20.00	-175.8	-0.997	-0.073
v _B =	-163.80	0.00	163.80	180.0	-1.000	0.000
v _A =	-176.78	176.78	250.00	135.0	-0.707	0.707
v _{BA} =	12.98	-176.78	177.25	-85.8	0.073	-0.997
a _B =	11095.70	0.00	11095.70	0.0	1.000	0.000
a _A =	8874.19	8803.48	12500.10	44.8	0.710	0.704
a _{BA} =	2221.51	-8803.48	9079.45	-75.8	0.245	-0.970

PROBLEM 2 (40 pts): On the following page a linkage has been drawn to scale. It is known that the link O_2A is rotating at the rate of $\omega = -10\frac{1}{s}$.

2a) Find all the instant centers associated with this linkage.

2b) Using the instant center graphical approach, determine the velocity of points C & P, the angular velocities of links O_4P & BC, and the angular velocity of the slider at A.

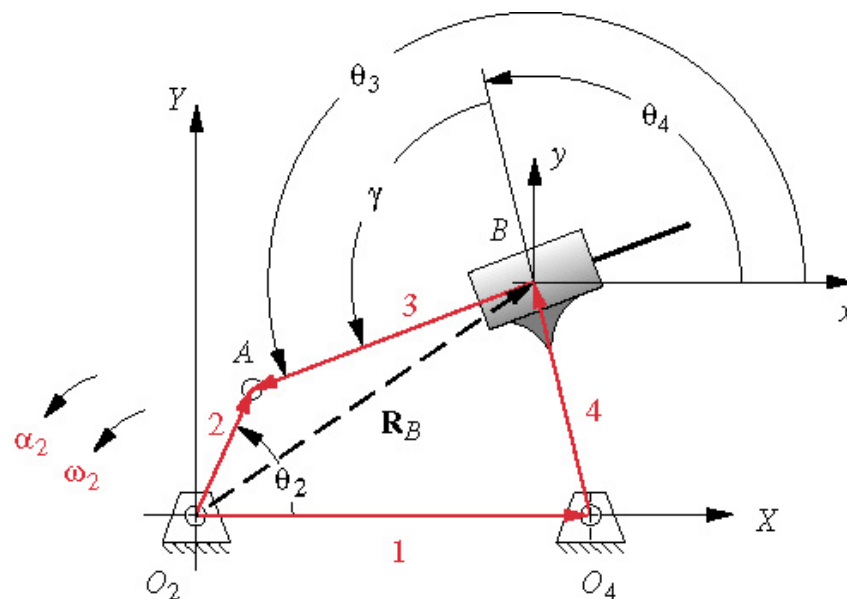




Bonus (5 pts): An inverted slider crank linkage has the following dimensions. All lengths are in inches and all angles are in degrees.

Link 1	Link 2	Link 4	θ_2	γ
5	8	8	150	90

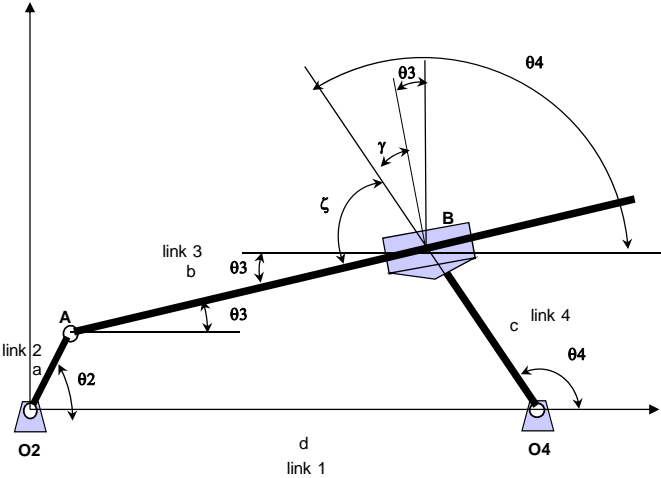
Link 2 is the drive link and it is known to be rotating at $\omega_2 = 100 \frac{1}{s}$ and $\alpha_2 = -65 \frac{1}{s^2}$. Using the program that you have been developing, calculate all the critical parameters associated with this linkage in both of the possible configurations. Print out the results of your program and staple it directly behind this page.



Inverted Slider Crank

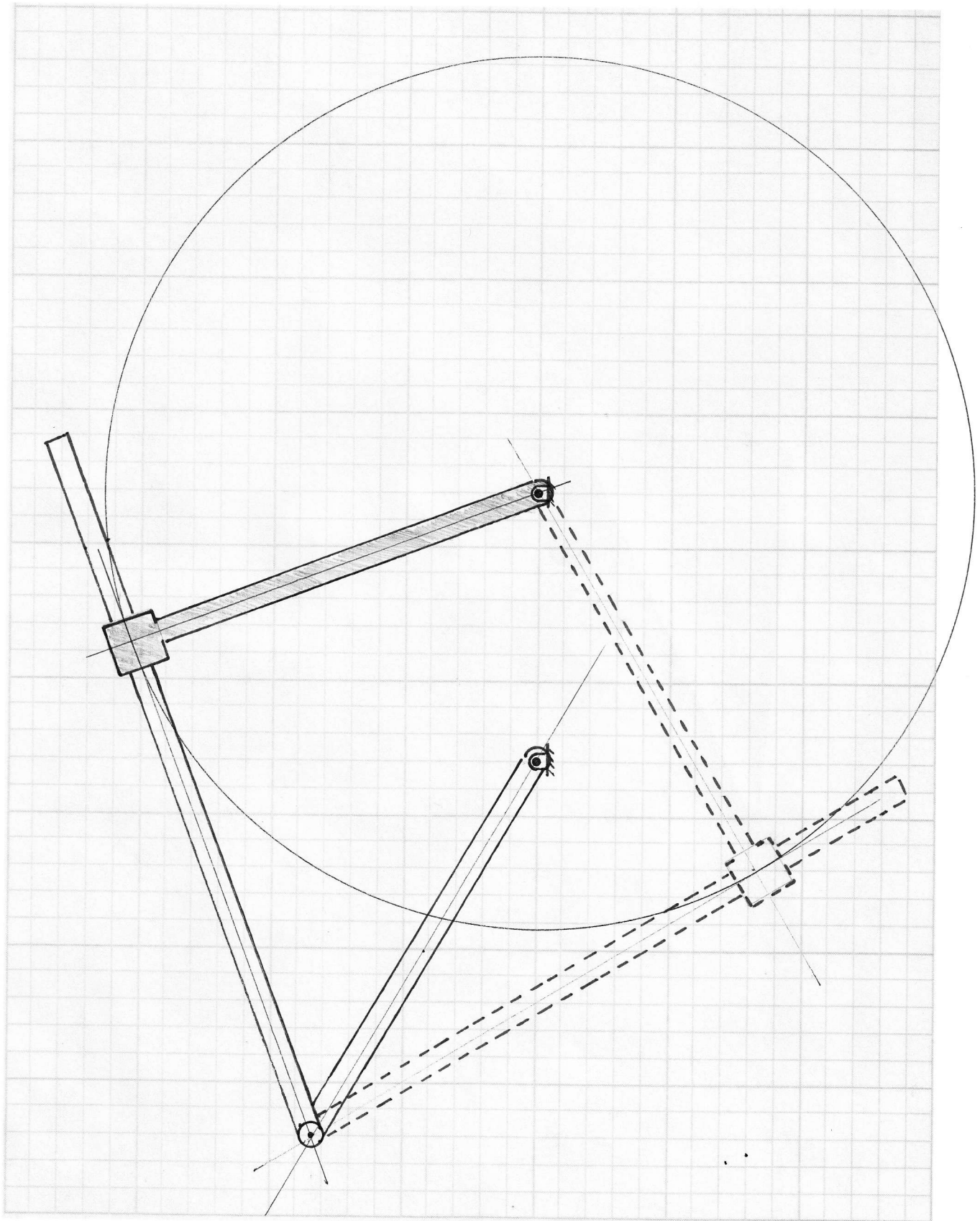
a=	8	Link 2
c=	8	Link 4
d=	5	Link 1
$\theta_2 =$	150	
$\dot{\gamma} =$	0	
$\dot{\theta}_2 =$	100	$1/s$
$\ddot{\theta}_2 =$	-65	$1/s^2$
b=	9.71	-9.71
$\theta_4 =$	110.95	-148.02
$\theta_3 =$	20.95	-238.02
$\dot{\theta}_4 =$	51.91	72.73
$\dot{\theta}_3 =$	51.91	72.73
b-dot=	205.98	-205.98
$\ddot{\theta}_4 =$	1942.01	-2645.70
$\ddot{\theta}_3 =$	1942.01	-2645.70
b-dotdot=	-40179.23	40179.23

$\zeta =$ 90
K1= -4
K2= 11.92820323
K3= 8



	x comp	y comp	mag	angle	i	j
rO4=	5.00	0.00	5.00	0.0	1.000	0.000
rA=	-6.93	4.00	8.00	150.0	-0.866	0.500
rBA=	9.07	3.47	9.71	20.9	0.934	0.358
rBO4=	-2.86	7.47	8.00	110.9	-0.358	0.934
rB=	2.14	7.47	7.77	74.0	0.275	0.961
vA=	-400.00	-692.82	800.00	-120.0	-0.500	-0.866
vBA=	12.17	544.36	544.50	88.7	0.022	1.000
vB=	-387.83	-148.46	415.27	-159.1	-0.934	-0.358
aA=	69542.03	-39549.67	80001.69	-29.6	0.869	-0.494
aBA=	-76344.88	13863.57	77593.42	169.7	-0.984	0.179
aB=	-6802.85	-25686.10	26571.68	-104.8	-0.256	-0.967
alt	x comp	y comp	mag	angle	i	j
rO4=	5.00	0.00	5.00	0.0	1.000	0.000
rA=	-6.93	4.00	8.00	150.0	-0.866	0.500
rBA=	5.14	-8.24	9.71	-58.0	0.530	-0.848
rBO4=	-6.79	-4.24	8.00	-148.0	-0.848	-0.530
rB=	-1.79	-4.24	4.60	-112.9	-0.388	-0.921
vA=	-400.00	-692.82	800.00	-120.0	-0.500	-0.866
vBA=	708.13	199.26	735.63	15.7	0.963	0.271
vB=	308.13	-493.56	581.84	-58.0	0.530	-0.848
aA=	69542.03	-39549.67	80001.69	-29.6	0.869	-0.494
aBA=	-44854.15	79913.90	91641.29	119.3	-0.489	0.872
aB=	24687.88	40364.24	47315.57	58.5	0.522	0.853

Bonus (5 pts): Draw the above inverted slider crank linkage in both of its configurations.



Bonus (5 pts): Draw the above inverted slider crank linkage in both of its configurations.

