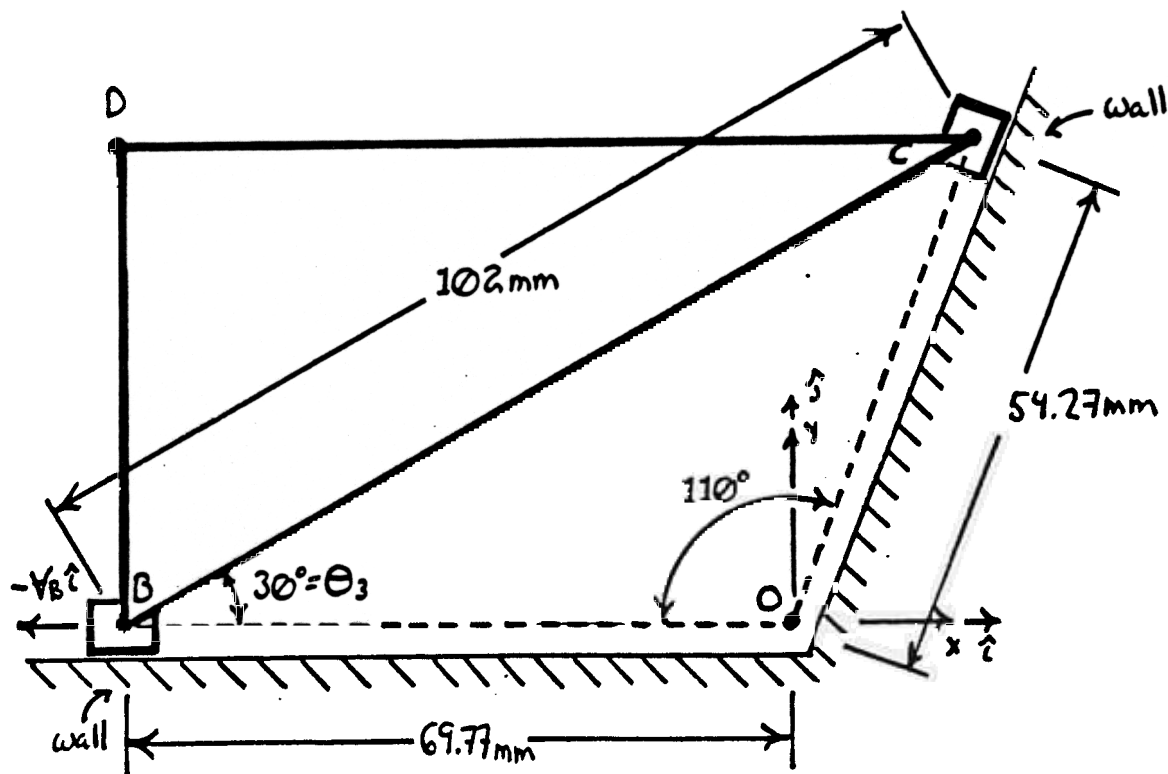


PROBLEM #1: Consider the mechanism shown in the figure below. The triangular wedge, coupler CDB is attached to two sliders at B and C. The joints at B and C are full joints. Both sliders are constrained to move along the wall frictionlessly. Point B is being forced to move at a constant velocity of 6.10 m/s to the left. For the position shown the loop closure equation is as follows:

where

$$\begin{aligned}\bar{\mathbf{R}}_{\text{BC}} &= \mathbf{R}_{\text{BC}} \cdot \mathbf{e}^{j\theta_3} = 102\text{mm} \cdot \mathbf{e}^{j30^\circ} = 88.33\text{mm} + j \cdot 51.0\text{mm} \\ &= \mathbf{R}_{\text{BC}} \cdot \hat{\mathbf{e}}_{\text{BC}} = 102\text{mm} \cdot (0.8660 \cdot \hat{\mathbf{i}} + 0.5 \cdot \hat{\mathbf{j}})\end{aligned}$$



- 1a) From a velocity analysis it was found that $\dot{\theta}_3 = -73.4 \frac{1}{s}$ and $\dot{R}_{oc} = -5.396 \frac{m}{s}$. differentiate the loop closure equation twice and determine \ddot{R}_{oc} using one of the analytical approaches.

Complex Method

$$R_{BO} e^{j\theta_1} + R_{oc} e^{j\theta_2} = R_{BC} e^{j\theta_3}$$

$$\dot{R}_{BO} e^{j\theta_1} + R_{BO} j \dot{\theta}_1 e^{j\theta_1} + \dot{R}_{oc} e^{j\theta_2} + R_{oc} j \dot{\theta}_2 e^{j\theta_2} = \dot{R}_{BC} e^{j\theta_3} + R_{BC} j \dot{\theta}_3 e^{j\theta_3}$$

$$\dot{R}_{BO} e^{j\theta_1} + \dot{R}_{oc} e^{j\theta_2} = R_{BC} j \dot{\theta}_3 e^{j\theta_3}$$

$$\ddot{R}_{BO} e^{j\theta_1} + \dot{R}_{BO} j \dot{\theta}_1 e^{j\theta_1} + \ddot{R}_{oc} e^{j\theta_2} + \dot{R}_{oc} j \dot{\theta}_2 e^{j\theta_2} = \ddot{R}_{BC} e^{j\theta_3} + R_{BC} j \ddot{\theta}_3 e^{j\theta_3} + R_{BC} j \dot{\theta}_3^2 e^{j\theta_3} - R_{BC} \dot{\theta}_3^2 e^{j\theta_3}$$

$$\ddot{R}_{oc} e^{j\theta_2} = R_{BC} j \ddot{\theta}_3 e^{j\theta_3} - R_{BC} \dot{\theta}_3^2 e^{j\theta_3}$$

$$\ddot{R}_{oc} (\cos\theta_2 + j \sin\theta_2) = R_{BC} j \ddot{\theta}_3 (\cos\theta_3 + j \sin\theta_3) - R_{BC} \dot{\theta}_3^2 (\cos\theta_3 + j \sin\theta_3)$$

$$\ddot{R}_{oc} \cos\theta_2 = -R_{BC} \ddot{\theta}_3 \sin\theta_3 - R_{BC} \dot{\theta}_3^2 \cos\theta_3$$

$$\ddot{R}_{oc} \sin\theta_2 = R_{BC} \ddot{\theta}_3 \cos\theta_3 - R_{BC} \dot{\theta}_3^2 \sin\theta_3$$

$$\ddot{R}_{oc} \cos\theta_2 \cos\theta_3 = -R_{BC} \ddot{\theta}_3 \sin\theta_3 \cos\theta_3 - R_{BC} \dot{\theta}_3^2 \cos\theta_3 \cos\theta_3$$

$$\ddot{R}_{oc} \sin\theta_2 \sin\theta_3 = R_{BC} \ddot{\theta}_3 \cos\theta_3 \sin\theta_3 - R_{BC} \dot{\theta}_3^2 \sin\theta_3 \sin\theta_3$$

$$\ddot{R}_{oc} \cos(\theta_2 - \theta_3) = -R_{BC} \ddot{\theta}_3$$

$$\ddot{R}_{oc} = \frac{-R_{BC} \dot{\theta}_3^2}{\cos(\theta_2 - \theta_3)} = \frac{(102 \text{ mm})(-73.4 \frac{1}{s})^2}{\cos(70^\circ - 30^\circ)} = -77.4 (10^3) \frac{\text{mm}}{\text{s}^2} = -77.4 \frac{\text{m}}{\text{s}^2}$$

Vector Method

$$R_{BO} \hat{e}_{BO} + R_{OC} \hat{e}_{OC} = R_{BC} \hat{e}_{BC}$$

$$\dot{R}_{BO} \hat{e}_{BO} + R_{BO} \dot{\hat{e}}_{BO} + \dot{R}_{OC} \hat{e}_{OC} + R_{OC} \dot{\hat{e}}_{OC} = \dot{R}_{BC} \hat{e}_{BC} + R_{BC} \dot{\hat{e}}_{BC}$$

$$\dot{R}_{BO} \hat{e}_{BO} + \dot{R}_{OC} \hat{e}_{OC} = R_{BC} \dot{\theta}_3 (\hat{h} \times \hat{e}_{BC})$$

$$\ddot{R}_{BO} \hat{e}_{BO} + \dot{R}_{BO} \dot{\hat{e}}_{BO} + \ddot{R}_{OC} \hat{e}_{OC} + \dot{R}_{OC} \dot{\hat{e}}_{OC} = \dot{R}_{BC} \dot{\theta}_3 (\hat{h} \times \hat{e}_{BC}) + R_{BC} \ddot{\theta}_3 (\hat{h} \times \hat{e}_{BC}) + R_{BC} \dot{\theta}_3^2 (\hat{h} \times (\hat{h} \times \hat{e}_{BC}))$$

$$\ddot{R}_{OC} \hat{e}_{OC} = R_{BC} \ddot{\theta}_3 (\hat{h} \times \hat{e}_{BC}) + R_{BC} \dot{\theta}_3^2 (\hat{h} \times (\hat{h} \times \hat{e}_{BC}))$$

Dotting with \hat{e}_{BC}

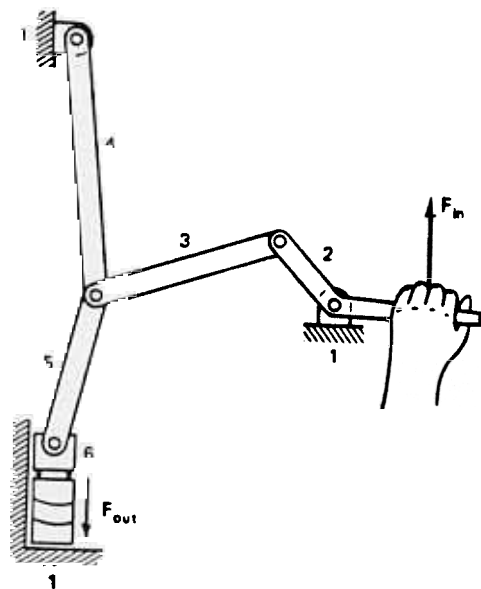
$$\ddot{R}_{OC} \hat{e}_{BC} \cdot \hat{e}_{OC} = R_{BC} \ddot{\theta}_3 \hat{e}_{BC} \cdot (\hat{h} \times \hat{e}_{BC}) + R_{BC} \dot{\theta}_3^2 \hat{e}_{BC} \cdot (\hat{h} \times (\hat{h} \times \hat{e}_{BC}))$$

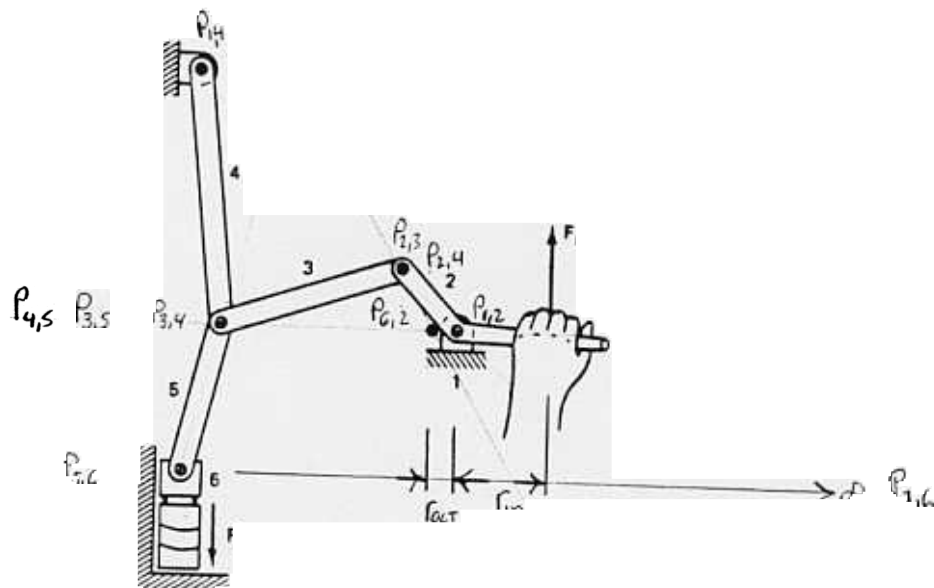
$$\ddot{R}_{OC} = R_{BC} \dot{\theta}_3^2 \frac{\hat{e}_{BC} \cdot (\hat{h} \times (\hat{h} \times \hat{e}_{BC}))}{\hat{e}_{BC} \cdot \hat{e}_{OC}}$$

$$\hat{e}_{BC} \cdot \hat{e}_{OC} = (-.860\hat{i} + .5\hat{j}) \cdot (.3420\hat{i} + .9397\hat{j}) = 0.7660$$

$$\ddot{R}_{OC} = 102\text{mm} (-73.4\frac{1}{3})^2 \frac{-1}{.7660} = 717.4(10^3) \frac{\text{mm}}{\text{s}^2} = 717.4 \frac{\text{m}}{\text{s}^2}$$

PROBLEM 2: For the mechanism shown, determine the mechanical advantage.





Knowing that

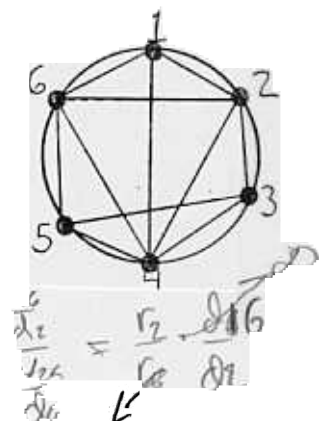
$$P \quad P_{out} \Rightarrow T_{in} \omega_{in} \quad F_{in} \quad v \quad F_{out} \quad v_{out} \quad T_{out} \omega_{out}$$

Because of the constraints in 6 I can state that $\theta_1 = \theta_{6,2}$

$$\begin{matrix} v_{in} & r_{in} & \omega_2 \\ v_{out} & r_{out} & \omega_2 \end{matrix} \rightarrow \begin{matrix} F_{in} & v_{in} & F_{out} & v_{out} \\ F_{in} & r_{in} & F_{out} & r_{out} \end{matrix}$$

$$MA = \frac{F_{out}}{F} \cdot \frac{r_{in}}{r_{out}} = \frac{15}{4} = \boxed{3.75}$$

$$MA = \frac{F_{out}}{F_{in}} = \frac{15}{4} = 3.75$$



PROBLEM 3: For a cam with the following characteristics:

- rise $\frac{3}{4}$ in with constant acceleration in 90° ,
- rise $\frac{3}{4}$ in with constant deceleration in 90° ,
- dwell 30° ,
- fall $\frac{3}{4}$ in with constant acceleration in 60° ,
- fall $\frac{3}{4}$ in with constant deceleration in 60° , and
- dwell 30° .

Determine the boundary conditions and order of the polynomial for each section of the cam.

Region #1 $0^\circ < \theta < 90^\circ$, $\beta = 90^\circ = \frac{\pi}{2} \text{ rad}$

$$S_1 = C_0 + C_1\left(\frac{\theta_1}{\beta_1}\right) + C_2\left(\frac{\theta_1}{\beta_1}\right)^2$$

$$\text{B.C.: } \theta_1 = 0, S_1(0) = 0 \text{ in}, v_1(0) = 0 \text{ in/rad} \\ = \frac{\pi}{2}, S_1\left(\frac{\pi}{2}\right) = \frac{3}{4} \text{ in}$$

Region 6 $330^\circ < \theta < 360^\circ$

$$0 < \theta_6 < 30^\circ$$

$$\beta = \frac{\pi}{6} = 30^\circ$$

$$S_6 = 0$$

Region #2 $90^\circ < \theta < 180^\circ$, $0 < \theta_2 < \frac{\pi}{2}$, $\beta = \frac{\pi}{2}$

$$S_2 = C_0 + C_1\left(\frac{\theta_2}{\beta_2}\right) + C_2\left(\frac{\theta_2}{\beta_2}\right)^2$$

$$\text{B.C. } \theta_2 = 0, S_2(0) = 0.75 \text{ in}$$

$$\theta_2 = \frac{\pi}{2}, S_2\left(\frac{\pi}{2}\right) = 1.5 \text{ in}, v_2\left(\frac{\pi}{2}\right) = 0 \text{ in/rad}$$

Region #3 $180^\circ < \theta < 210^\circ$, $0 < \theta_3 < \frac{\pi}{6}$, $\beta = \frac{\pi}{6}$

$$S_3 = 1.5 \text{ in}$$

Region #4 $210^\circ < \theta < 270^\circ$, $0 < \theta_4 < \frac{\pi}{3}$, $\beta = \frac{\pi}{3}$

$$S_4 = C_0 + C_1\left(\frac{\theta_4}{\beta_4}\right) + C_2\left(\frac{\theta_4}{\beta_4}\right)^2$$

$$\text{B.C.: } \theta_4 = 0, S_4(0) = 1.5 \text{ in}, v_4(0) = 0 \text{ in/rad}$$

$$\theta_4 = \frac{\pi}{3}, S_4\left(\frac{\pi}{3}\right) = 0.75 \text{ in}$$

Region #5 $270^\circ < \theta < 330^\circ$, $0 < \theta_5 < \frac{\pi}{3}$, $\beta = \frac{\pi}{3}$

$$S_5 = C_0 + C_1\left(\frac{\theta_5}{\beta_5}\right) + C_2\left(\frac{\theta_5}{\beta_5}\right)^2$$

$$\text{B.C.: } \theta_5 = 0, S_5(0) = 0.75 \text{ in}$$

$$S_5\left(\frac{\pi}{3}\right) = 0 \text{ in}, v_5\left(\frac{\pi}{3}\right) = 0 \text{ in/rad}$$

EXTRA CREDIT: Draw the SVAJ diagram for this cam. Note, it is not necessary to solve the polynomial for each section of the cam.

