

Name: Solution

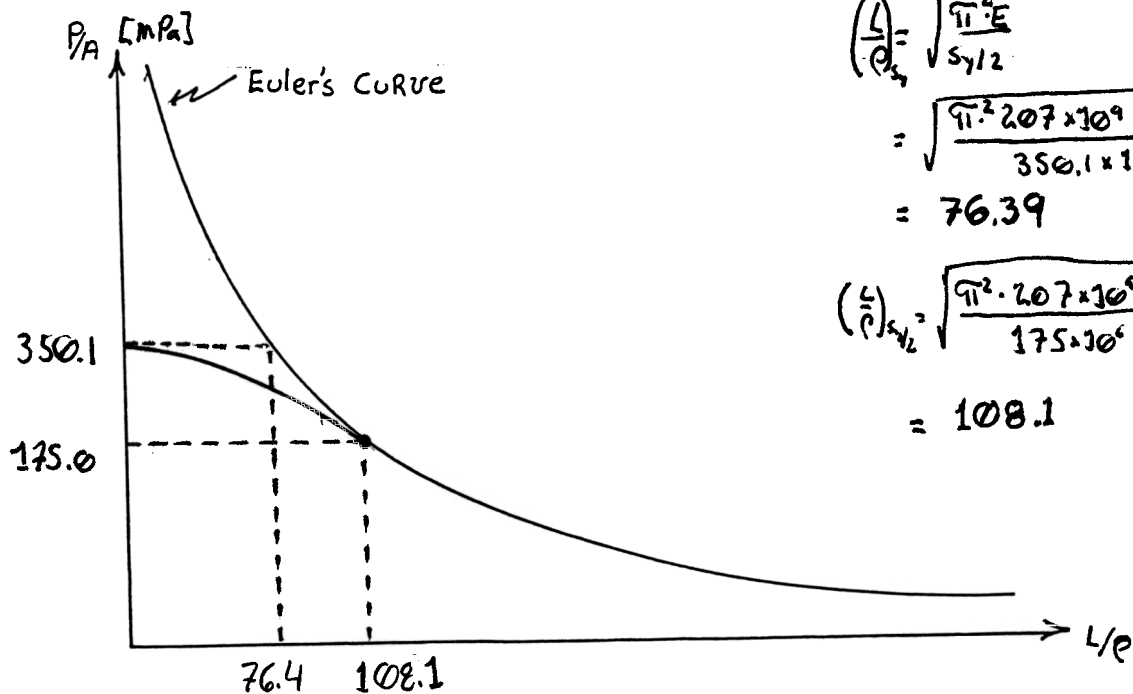
**Problem 1.** A 20 mm diameter steel rod of  $S_y = 350$  MPa is loaded as a column with pinned ends. If it is sufficiently short, it can carry a limiting load of  $S_y \bullet A = 110$  kN.

**1a.** Using the figure provided, draw the  $P/A$  and  $L/\rho$  locations for the location on Euler's curve where Euler's Equation is no longer valid. Also, draw the Johnson (Parabolic) curve on this figure being sure to label the critical  $P/A$  and  $L/\rho$  locations. Where

$$E = 207 \text{ GPa} \quad \rho = \sqrt{I/A} = d/4 = 5 \text{ mm}$$

$$A = \pi \frac{D^2}{4} = \pi \frac{(20 \times 10^{-3} \text{ m})^2}{4} = 0.3142 (10^{-3}) \text{ m}^2$$

$$S_y = 350.1 (10^6) \text{ Pa} \\ 350.1 \text{ MPa}$$



$$\left(\frac{L}{\rho}\right)_y = \sqrt{\frac{\pi^2 E}{S_y/2}} \\ = \sqrt{\frac{\pi^2 \cdot 207 \times 10^9 \text{ N/m}^2}{350.1 \times 10^6 \text{ N/m}^2}} \\ = 76.39$$

$$\left(\frac{L}{\rho}\right)_{S_y/2} = \sqrt{\frac{\pi^2 \cdot 207 \times 10^9 \text{ N/m}^2}{175.05 \times 10^6 \text{ N/m}^2}} \\ = 108.1$$

1b. How long can the rod be and still carry 10% of the limiting load?

$$10\% \cdot 350.1 \text{ MPa} = 35.01 \text{ MPa}$$

$$\left(\frac{L}{\rho}\right) = \sqrt{\frac{\pi^2 \cdot 207.10^{9} \text{ N/m}^2}{35.01 \times 10^6 \text{ N/m}^2}} = 241.6 \quad \Rightarrow \quad L = 241.6 \cdot 5 \times 10^3 \text{ m}$$

$$= 1.208 \text{ m}$$

$$\boxed{1.21 \text{ m}}$$

1c. How long can the rod be and still carry 90% of the limiting load?

$$.90 \cdot 350.1 \text{ MPa} = 315.1 \text{ MPa}$$

$$\left(\frac{l}{\rho}\right) = \sqrt{\frac{\pi^2 \cdot 207 \times 10^9 \text{ N/m}^2}{315.1 \times 10^6 \text{ N/m}^2}} = 80.52$$

This places the beam in an intermediate range. Johnson's equation is required.

$$\frac{P_{cr}}{A} = S_y - \frac{S_y^2}{4\pi^2 E} \left(\frac{l}{\rho}\right)^2$$

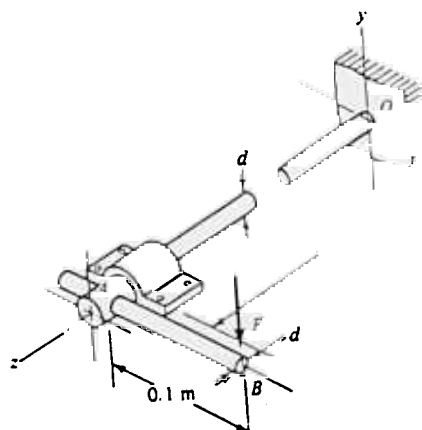
$$315.1 \times 10^6 \text{ N/m}^2 = 350.1 \times 10^6 \text{ N/m}^2 - \frac{(350.1 \times 10^6 \text{ N/m}^2)^2}{4 \times \pi^2 \cdot 207 \times 10^9 \text{ N/m}^2} \cdot \left(\frac{l}{\rho}\right)^2$$

$$\frac{l}{\rho} = 48.31 \Rightarrow l = 48.31 \cdot 5 \times 10^{-3} \text{ m} = 0.2416 \text{ m} = \boxed{0.242 \text{ m}}$$

**Problem 2.** Illustrated below is a torsion-bar spring OA having a diameter  $d=12\text{mm}$ . The actuating cantilever AB also has  $d=12\text{mm}$ . Both parts are made of carbon steel.

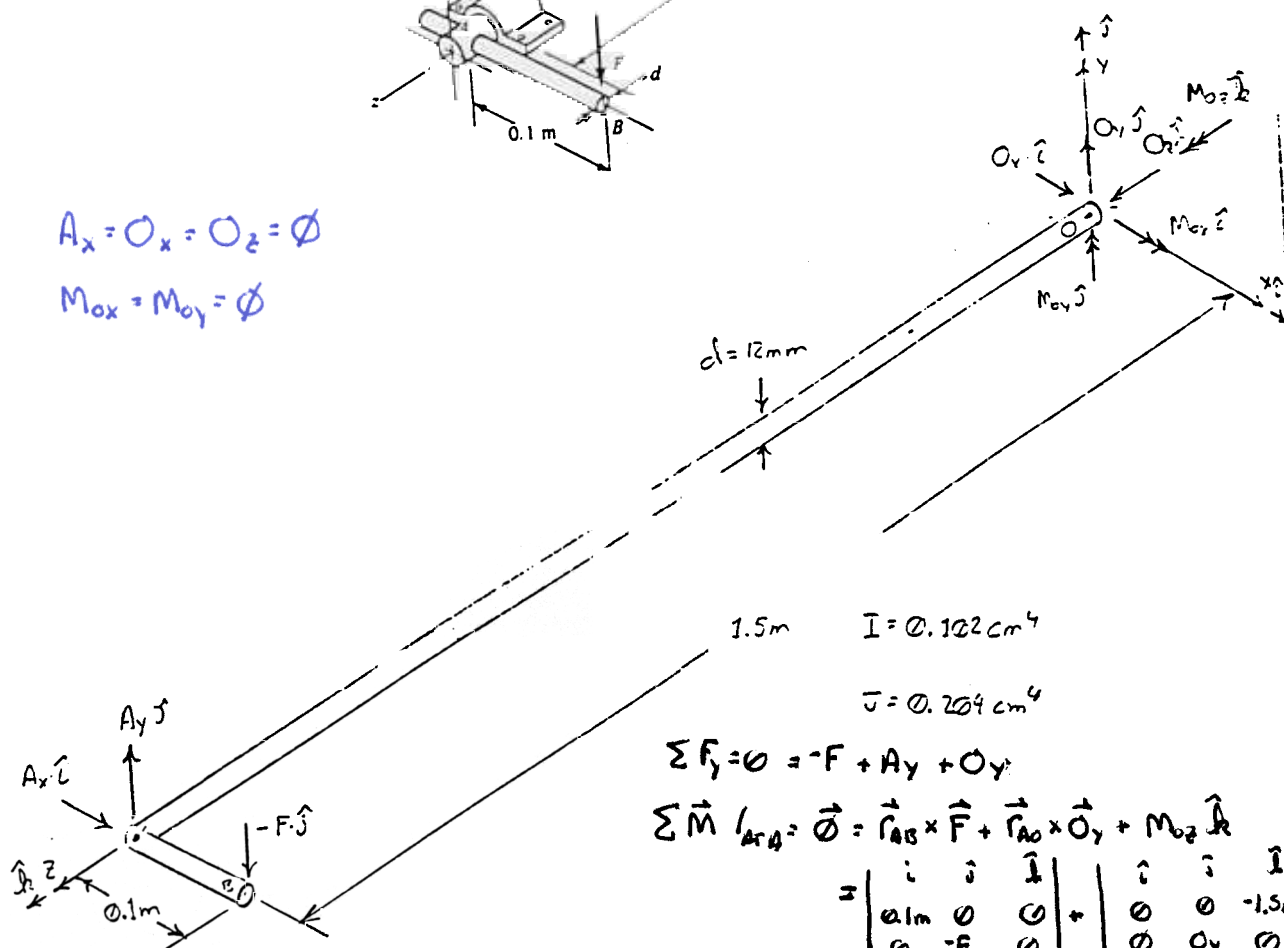
$$G=79.3 \text{ GPa}, E=207 \text{ GPa}, I=0.102\text{cm}^4, J=0.204\text{cm}^4$$

Using Castiglione's Theorem, find the spring force  $k$  corresponding to a force acting at B.



$$A_x = O_x = O_z = 0$$

$$M_{ox} = M_{oy} = 0$$



1.5 m

$$I = 0.102 \text{ cm}^4$$

$$J = 0.204 \text{ cm}^4$$

$$\sum F_y = 0 = -F + A_y + O_y$$

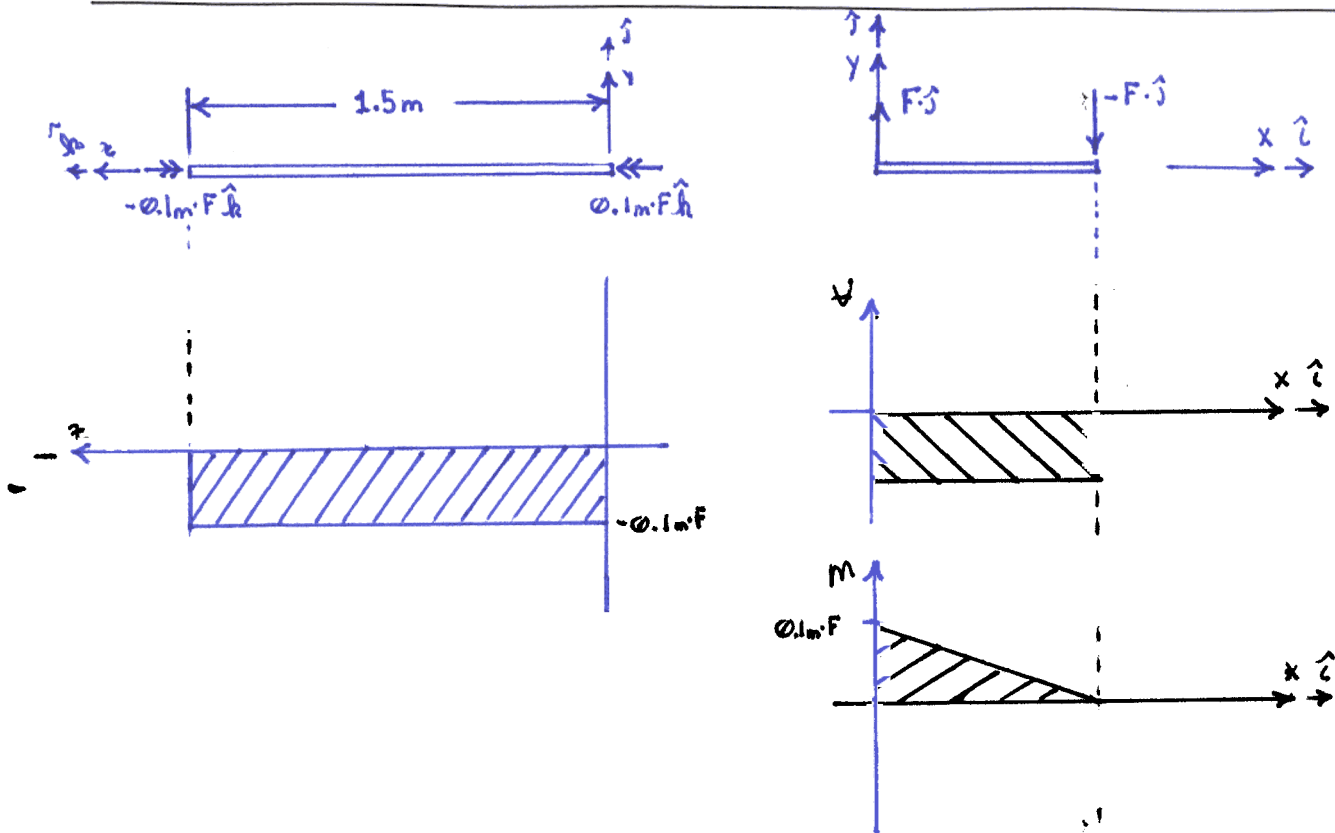
$$\sum \vec{M} /_{A/O} = \vec{0} = \vec{r}_{AB} \times \vec{F} + \vec{r}_{AO} \times \vec{O}_y + M_{Oz} \hat{k}$$

$$= \begin{vmatrix} \hat{i} & \hat{j} & \hat{k} \\ 0.1\text{m} & 0 & 0 \\ 0 & -F & 0 \end{vmatrix} + \begin{vmatrix} \hat{i} & \hat{j} & \hat{k} \\ 0 & 0 & -1.5\text{m} \\ 0 & O_y & 0 \end{vmatrix} + M_{Oz} \hat{k}$$

$$= -0.1\text{m} \cdot F \hat{k} + 1.5\text{m} \cdot O_y \hat{i} + M_{Oz} \hat{k}$$

$$\sum M_{y/A} = 0 = 1.5\text{m} \cdot O_y \Rightarrow \boxed{O_y = 0} \Rightarrow \boxed{A_y = F}$$

$$\sum M_{z/A} = 0 = -0.1\text{m} \cdot F + M_{Oz} \Rightarrow \boxed{M_{Oz} = 0.1\text{m} \cdot F}$$



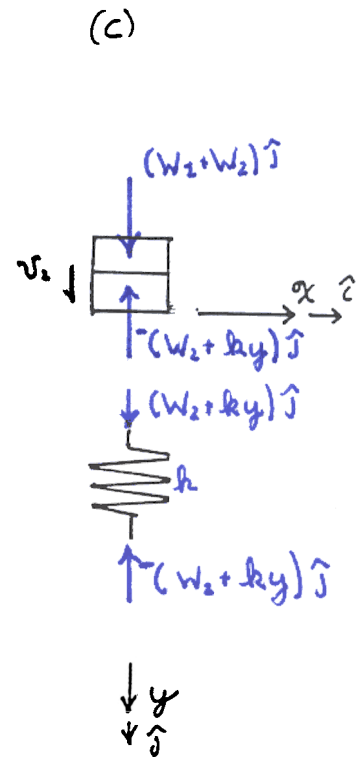
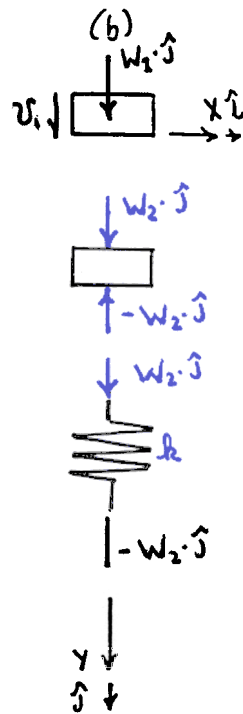
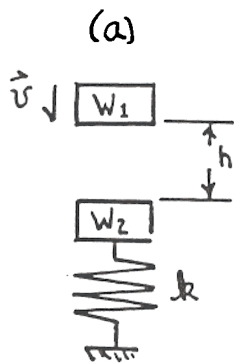
$$\begin{aligned}
 U &= \int_0^{1.5m} \frac{I^2}{2 \cdot E \cdot J} dz + \int_0^{0.1m} \frac{M^2}{2 \cdot E \cdot I} dx = \int_0^{1.5m} \frac{(1+V)^2 \cdot I^2}{E \cdot J} dz + \int_0^{0.1m} \frac{M^2}{2 \cdot E \cdot I} dx \\
 &= \int_0^{1.5m} \frac{0.01 m^2 \cdot F^2}{2 \cdot (79.3 \times 10^9 N/m^2) \cdot (0.204 \times 10^{-8} m^4)} dz + \int_0^{0.1m} \frac{(-x + 0.1m)^2 \cdot F^2}{2 \cdot (207 \times 10^9 N/m^2) \cdot (0.101 \times 10^{-8} m^4)} dx \\
 &= \int_0^{1.5m} \frac{0.01 m^2 \cdot F^2}{323.5 N \cdot m^2} dz + \int_0^{0.1m} \frac{(-x + 0.1m)^2}{422.3 N \cdot m^2} dx \\
 \frac{\partial U}{\partial F} &= \int_0^{1.5m} \frac{0.02 m^2 \cdot F}{323.5 N \cdot m^2} dz + \int_0^{0.1m} \frac{(2x^2 - 0.4m \cdot x + 0.02m^2) \cdot F}{422.3 N \cdot m^2} dx \\
 &= \frac{0.02 m^2 \cdot F \cdot z}{323.5 N \cdot m^2} \bigg|_0^{1.5m} + \frac{F}{422.3 N \cdot m^2} \left( \frac{2}{3} x^3 - 0.2m \cdot x^2 + 0.02m^2 \cdot x \right) \bigg|_0^{0.1m}
 \end{aligned}$$

$$\delta = \frac{\partial U}{\partial F} = 94.31(10^{-6}) \frac{N}{m} \cdot F \Rightarrow F = 10.60 \times 10^3 \frac{N}{m} \cdot \delta$$

$$k = \frac{F}{\delta} = 10.6 (10^3) \frac{N}{m} = \boxed{10.6 \frac{kN}{m}}$$

**Problem 3.** As shown in the figure, the weight  $W_1$  strikes  $W_2$  from a height  $h$ .

**3a.** Figure b is an incomplete free-body-diagram of the system prior to  $W_1$  impacting  $W_2$  and Figure c is an incomplete free-body-diagram of the system after  $W_1$  has impacted  $W_2$ . Complete these free-body-diagrams.



3b. Impose equilibrium on the free-body-diagram found in Figure c and determine the differential equation that represents the motion of the system. What are the initial conditions for this system after  $W_1$  and  $W_2$  have impacted.

$$\sum \vec{F} = m \vec{a}$$

Dotting the above equation with  $\hat{j}$

$$(W_1 + W_2) - (W_2 + k \cdot y) = m \cdot \ddot{y} = \frac{W_1 + W_2}{g} \cdot \ddot{y}$$

$$W_1 - k \cdot y = \frac{W_1 + W_2}{g} \ddot{y}$$

The initial conditions are

$$t=0, y=0$$

$$t=0, \dot{y} = v^2 = \frac{W_1}{W_1 + W_2} \cdot v_1 = \frac{W_1}{W_1 + W_2} \cdot \sqrt{2 \cdot g \cdot h}$$