

NAME: **SOLUTION**

PROBLEM 1: Given the state of stress

$$[\sigma] = \begin{bmatrix} 40 & 40 & 30 \\ 40 & 20 & 0 \\ 30 & 0 & 20 \end{bmatrix} MPa$$

a. Determine the stress invariants.

Using MatLab

```
STRE=[40 40 30; 40 20 0; 30 0 20]
```

```
STRE =
```

```
40 40 30
40 20 0
30 0 20
```

```
poly(STRE)
```

```
ans = 1.0e+004 * (σ³ -0.0080 σ² -0.0500 σ³ + 3.4000)
```

I₁=80MPa I₂=-500MPa² I₃=-34,000MPa³

b. Determine the principal stresses.

Using MatLab

```
>> [V P]=eig(STRE)
```

```
V =
```

```
-0.6340  0.0000  0.7733
 0.6187 -0.6000  0.5072
 0.4640  0.8000  0.3804
```

```
P =
```

```
-20.9902    0    0
    0 20.0000    0
    0    0 80.9902
```

σ₁=-21 MPa σ₂=20 MPa σ₃=81 MPa
--

- c. Determine the direction cosines to each of the principal stresses and calculate $\theta_{x'x}$, $\theta_{x'y}$, $\theta_{x'z}$, $\theta_{y'x}$, $\theta_{y'y}$, $\theta_{y'z}$, $\theta_{z'x}$, $\theta_{z'y}$, and $\theta_{z'z}$.
Using MatLab

```
>> acos(V)*180/pi
```

```
ans =
```

```
129.3450  90.0000  39.3450
51.7807  126.8699  59.5231
62.3541  36.8699  67.6420
```

$\theta_{x'x}=129.3$	$\theta_{x'y}=51.8$	$\theta_{x'z}=62.4$
$\theta_{y'x}=90.0$	$\theta_{y'y}=126.9$	$\theta_{y'z}=36.9$
$\theta_{z'x}=39.3$	$\theta_{z'y}=59.5$	$\theta_{z'z}=67.6$

- d. Determine the transformation matrix from the original state of stress to the principal state of stress and prove that it is the transformation matrix by using it to transform the original state of stress.

```
>> T=V'
```

```
T =
```

-0.6340	0.6187	0.4640
0.0000	-0.6000	0.8000
0.7733	0.5072	0.3804

```
>> T*STRE*T'
```

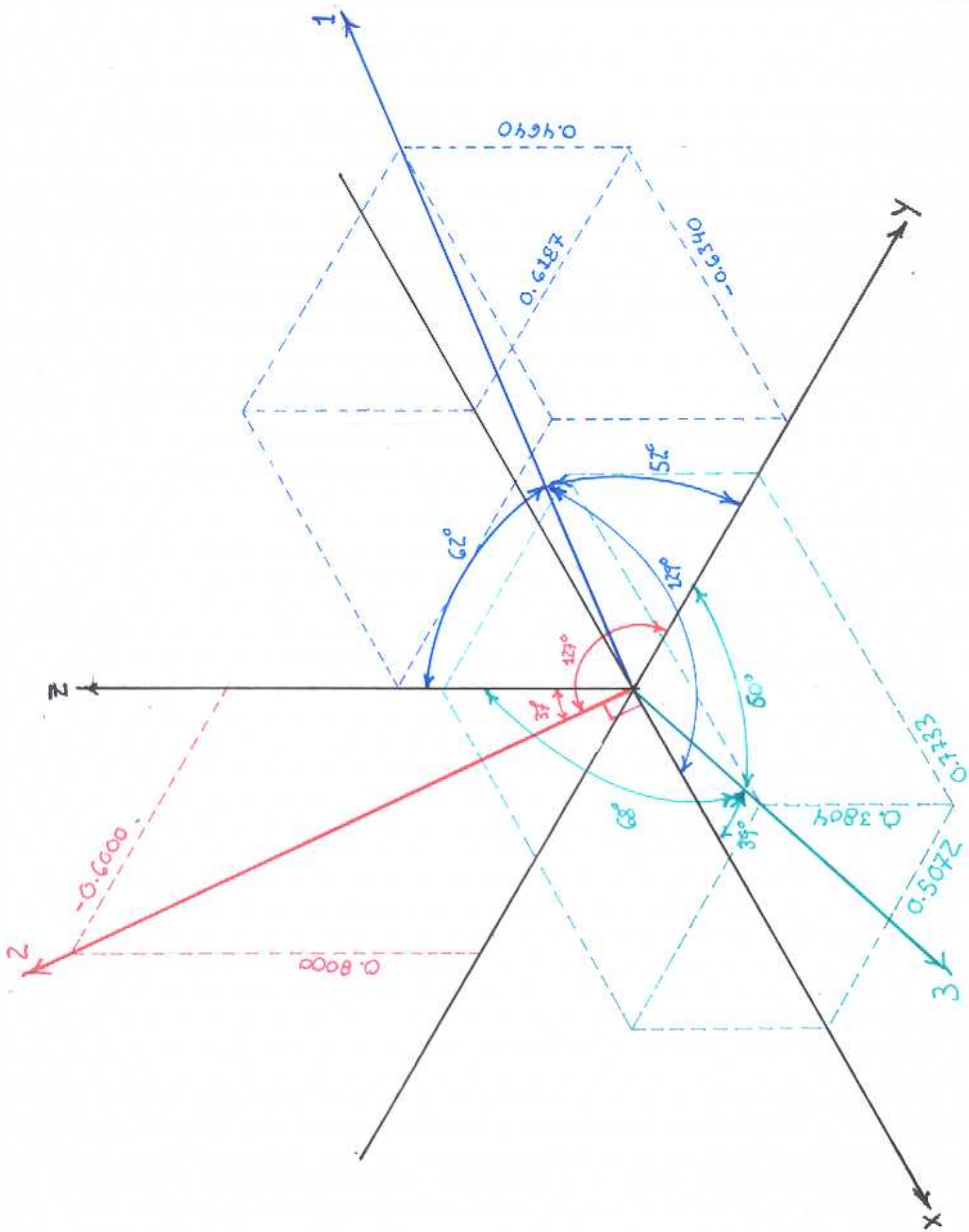
```
ans =
```

```
-20.9902  -0.0000  -0.0000
-0.0000  20.0000   0.0000
0.0000   0.0000  80.9902
```

```
>> P
```

```
P =
```

```
-20.9902    0    0
    0  20.0000    0
    0    0  80.9902
```



- e. Determine the state of stress defined by rotating x,y plane in the original state of stress through an angle of 30° clockwise about the z axis.

Using MatLab

```
>> T2=[0.866 -.5 0; .5 0.866 0; 0 0 1]
```

T2 =

```
0.8660 -0.5000    0
0.5000  0.8660    0
0        0    1.0000
```

```
>> T2*STRE*T2'
```

ans =

```
0.3582 28.6582 25.9800
28.6582 59.6391 15.0000
25.9800 15.0000 20.0000
```

$\sigma_x = 0.4 \text{ MPa}$	$\tau_{x'y'} = 28.7 \text{ MPa}$	$\tau_{x'z'} = 26.0 \text{ MPa}$
$\tau_{x'y'} = 28.7 \text{ MPa}$	$\sigma_{y'} = 60.0 \text{ MPa}$	$\tau_{y'z'} = 15.0 \text{ MPa}$
$\tau_{x'z'} = 26.0 \text{ MPa}$	$\tau_{y'z'} = 15.0 \text{ MPa}$	$\sigma_{z'} = 20.0 \text{ MPa}$

- f. Determine the maximum shear stress for this state of stress.

Using MatLab

```
>> (P(1,1)-P(3,3))/2
```

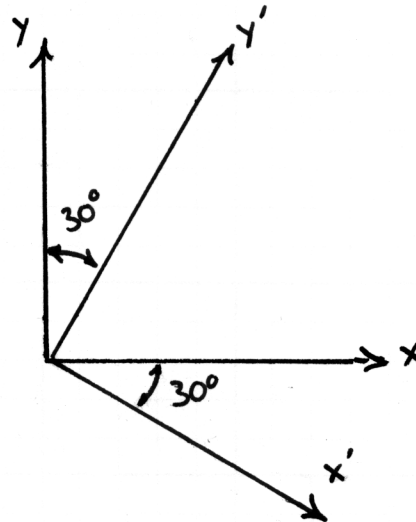
ans = -50.9902

$\tau_{\max} = 51 \text{ MPa}$

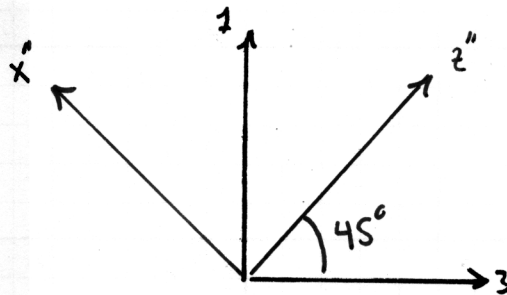
TRANSFORMATION MATRIX FOR A 30° CLOCKWISE ROTATION ABOUT THE Z-AXIS.

$$[T] = \begin{bmatrix} \cos \theta_{xx} & \cos \theta_{xy} & \cos \theta_{xz} \\ \cos \theta_{yx} & \cos \theta_{yy} & \cos \theta_{yz} \\ \cos \theta_{zx} & \cos \theta_{zy} & \cos \theta_{zz} \end{bmatrix}$$

$$= \begin{bmatrix} 0.866 & -0.5 & 0 \\ 0.5 & 0.866 & 0 \\ 0 & 0 & 1 \end{bmatrix}$$



TRANSFORMING THE PRINCIPAL STATE OF STRESS 45° ABOUT THE Z-AXIS



$$[T] = \begin{bmatrix} \cos \theta_{x'1} & \cos \theta_{x'2} & \cos \theta_{x'3} \\ \cos \theta_{y'1} & \cos \theta_{y'2} & \cos \theta_{y'3} \\ \cos \theta_{z'1} & \cos \theta_{z'2} & \cos \theta_{z'3} \end{bmatrix} = \begin{bmatrix} 0.7071 & 0 & -0.7071 \\ 0 & 1 & 0 \\ 0.7071 & 0 & 0.7071 \end{bmatrix}$$

- g. Determine the transformation matrix that needs to be used to transform the original state of stress to a state of stress that contains the maximum shear stress on two of the faces and a principal state of stress on the third.**

To get to the maximum shear stress state, the original stress state is transformed to the principal state of stress and then the principal state of stress is rotated 45° about the 2-axis. The transformation is [T3]

>> T3=[.7071 0 -.7071; 0 1 0; .7071 0 .7071]

T3 =

```
0.7071    0 -0.7071
    0 1.0000    0
0.7071    0  0.7071
```

The transformation matrix from the original state of stress to the state of stress where one surface is in the principal state and on the other two surfaces the maximum shear stress exists can not be calculated.

>> T4=T3*T

T4 =

-0.9951	0.0788	0.0591
0.0000	-0.6000	0.8000
0.0985	0.7961	0.5971

This result is proven by first calculating the desired state of stress through a transformation to the principal state of stress and then a second transformation about the 2-axis to the state of maximum shear.

>> T3*T*STRE*T'*T3'

ans =

```
29.9994 -0.0000 -50.9892
-0.0000 20.0000  0.0000
-50.9892  0.0000 29.9994
```

This is compared to the single calculation that was calculated above.

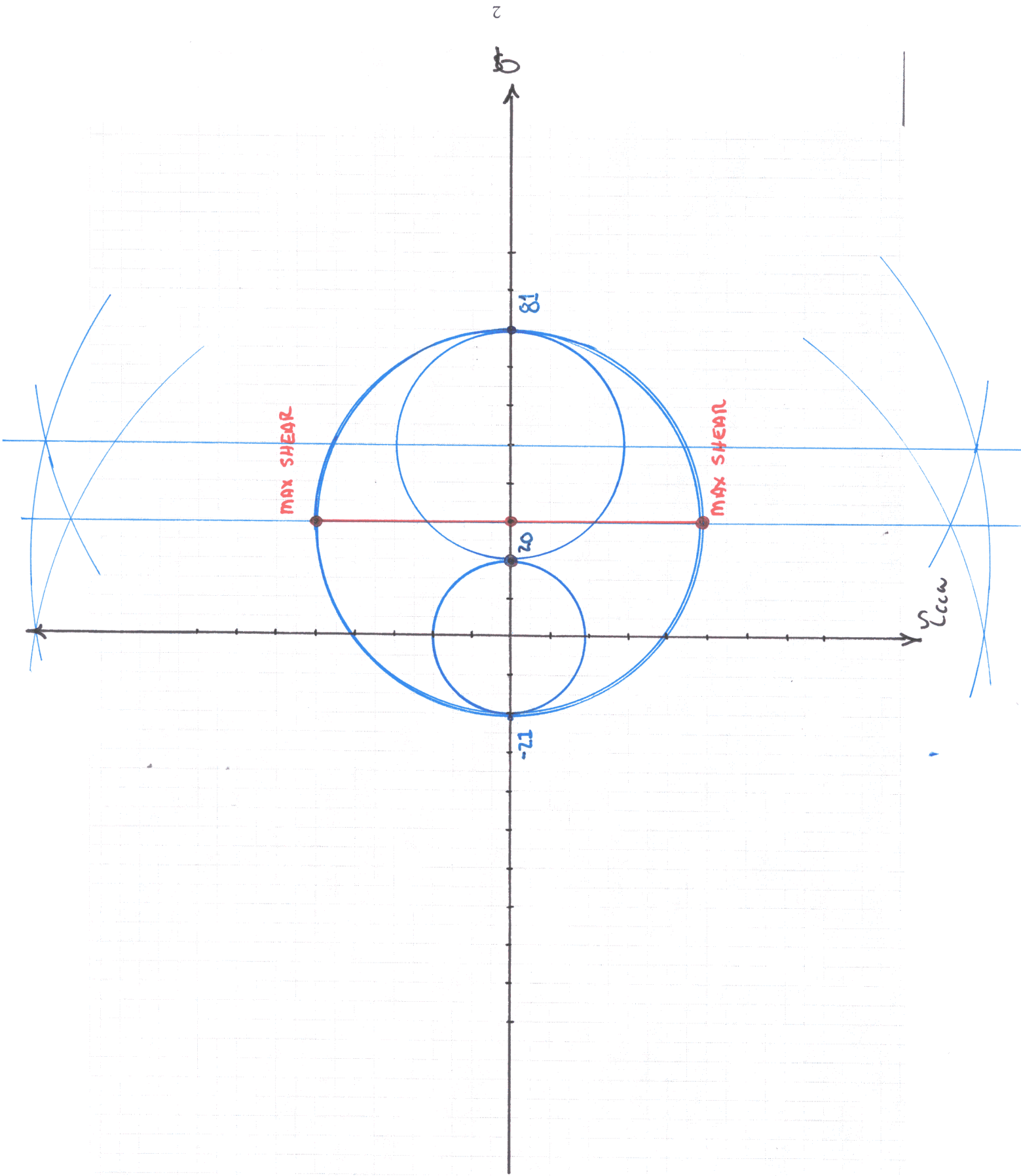
>> T4*STRE*T4'

ans =

```
29.9994 -0.0000 -50.9892
-0.0000 20.0000  0.0000
-50.9892  0.0000 29.9994
```

The two solutions match; therefore, [T4] is the desired transformation matrix

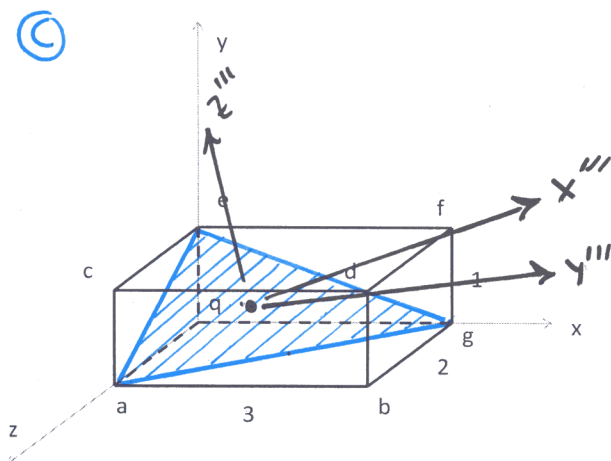
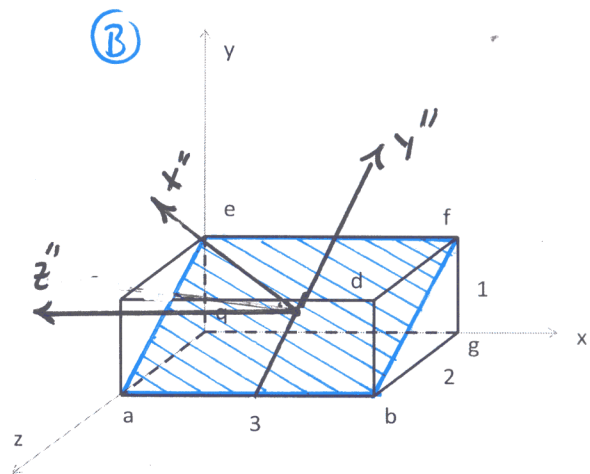
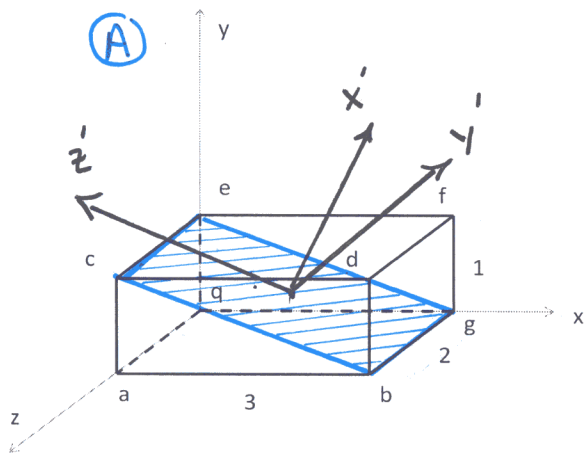
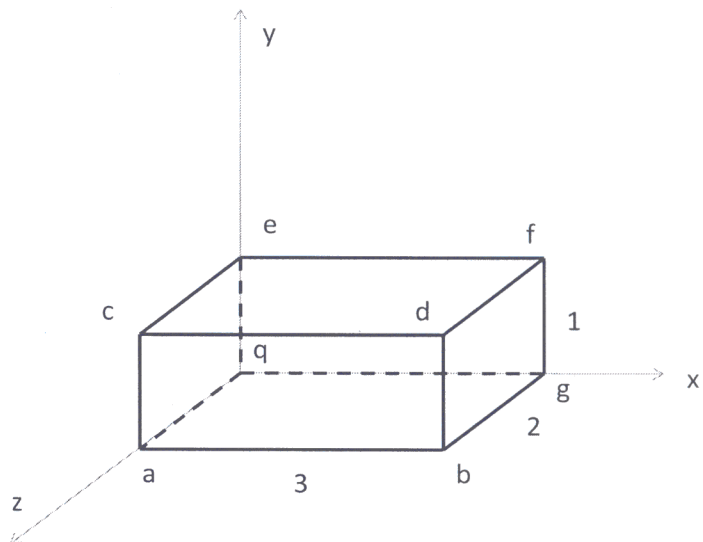
- h. Draw the Mohr's circle that defines the bounds for this state of stress.**



PROBLEM 3: Determine the transformation matrix for rotating a state of stress on the cube shown to the surface parallel to the following surfaces:

- CEBG
- ABEF
- AEG

Be sure to describe your justification for each of the coordinates used. Draw the transformed coordinate system with respect to the cube.



The transformation matrix for CEBG

```
>> RAbg=[0 0 -2]

RAbg =
    0    0   -2
>> RAbc=[-3 1 0]
RAbc =
   -3    1    0

>> N=cross(RAbg,RAbc)
N =
    2    6    0

>> Uxp=N/norm(N)
Uxp =
    0.3162    0.9487     0

>> Uzp=RAbc/norm(RAbc)
Uzp =
   -0.9487    0.3162     0

>> Uyp=cross(Uzp,Uxp)
Uyp =
    0     0   -1.0000

>> Tp=[Uxp; Uyp; Uzp]
```

Tp =			
0.3162	0.9487	0	
0	0	-1.0000	
-0.9487	0.3162	0	

The transformation matrix for ABEF

```
>> RBbf=[0 1 -2]
RBbf =
    0    1   -2

>> RBba=[-3 0 0]
RBba =
   -3    0    0

>> Npp=cross(RBbf,RBba)
Npp =
    0    6    3

>> Uxpp=Npp/norm(Npp)
Uxpp =
    0  0.8944  0.4472

>> Uypp=RBbf/norm(RBbf)
Uypp =
    0  0.4472 -0.8944

>> Uzpp=RBba/norm(RBba)
Uzpp =
   -1    0    0

>> Tpp=[Uxpp; Uypp; Uzpp]
Tpp =
    0  0.8944  0.4472
    0  0.4472 -0.8944
   -1.0000    0    0
```

The transformation matrix for AEG

```
>> RCag=[3 0 -2]
RCag =
    3    0   -2

>> RCae=[0 1 -2]
RCae =
    0    1   -2

>> Nppp=cross(RCag,RCae)
Nppp =
    2    6    3

>> Uxppp=Nppp/norm(Nppp)
Uxppp =
    0.2857    0.8571    0.4286

>> Uyppp=RCag/norm(RCag)
Uyppp =
    0.8321     0   -0.5547

>> Uzppp=cross(Uxppp,Uyppp)
Uzppp =
   -0.4755    0.5151   -0.7132

>> Tppp=[Uxppp; Uyppp; Uzppp]
Tppp =
    0.2857    0.8571    0.4286
    0.8321     0   -0.5547
   -0.4755    0.5151   -0.7132
```

PROBLEM 3: For the state of stress in Problem 1, determine the state of strain given $E=70\text{GPa}$ and $\nu=0.3$.

```
>> C=[(1/70e9) (-.3/70e9) (-.3/70e9) 0 0 0;
(-.3/70e9) (1/70e9) (-.3/70e9) 0 0 0;
(-.3/70e9) (-.3/70e9) (1/70e9) 0 0 0;
0 0 0 (2*(1+.3)/70e9) 0 0;
0 0 0 0 (2*(1+.3)/70e9) 0;
0 0 0 0 0 (2*(1+.3)/70e9)]
```

```
C =
1.0e-010 *
    0.1429   -0.0429   -0.0429         0         0         0
   -0.0429    0.1429   -0.0429         0         0         0
   -0.0429   -0.0429    0.1429         0         0         0
         0         0         0    0.3714         0         0
         0         0         0         0    0.3714         0
         0         0         0         0         0    0.3714
```

```
>> S=inv(C)
```

```
S =
1.0e+010 *
    9.4231    4.0385    4.0385         0         0         0
    4.0385    9.4231    4.0385         0         0         0
    4.0385    4.0385    9.4231         0         0         0
         0         0         0    2.6923         0         0
         0         0         0         0    2.6923         0
         0         0         0         0         0    2.6923
```

```
>> VSTRE=[40e6 20e6 20e6 0 30e6 40e6]'
```

```
VSTRE =
40000000
20000000
20000000
         0
30000000
40000000
```

```
>> VSTRA=C*VSTRE
```

```
VSTRA =
0.0004000
0.0000286
0.0000286
         0
0.001143
0.0014857
```

$\epsilon_x = 400 \mu\epsilon$ $\epsilon_y = 29 \mu\epsilon$ $\epsilon_z = 29 \mu\epsilon$ $\gamma_{xy} = 0$ $\gamma_{xz} = 1143 \mu\epsilon$ $\gamma_{yz} = 1486 \mu\epsilon$
