

PROBLEM 2.17 FOR THE PLANE STRESS STATE: 40 MPa , -10 MPa , 0 MPa , $\theta = -15^\circ$

- DRAW THE CORRESPONDING STRESS ELEMENT PROPERLY ORIENTED RELATIVE TO THE xy AXIS
- DETERMINE THE COMPLETE STRESS ELEMENT ASSOCIATED WITH AN AXIS SYSTEM ROTATED θ (DEFINED POSITIVE COUNTERCLOCKWISE) USING THE TRANSFORMATION EQUATIONS ALONG.
- DETERMINE THE PRINCIPAL STRESSES AND THE CORRESPONDING STRESS ELEMENT CONTAINING THE STRESSES PROPERLY ORIENTED RELATIVE TO THE xy AXES USING EQUATIONS ONLY.
- REPEAT PARTS (b) AND (c) USING MOHR'S CIRCLE.
- DETERMINE THE MAXIMUM AND MINIMUM SHEAR STRESS AND SHOW THE COMPLETE STRESS ELEMENT CONTAINING THESE STRESSES. SHOW THE ELEMENT PROPERLY ORIENTED WITH RESPECT TO THE xyz COORDINATE SYSTEM

GIVEN:

CONSTRAINTS

1. $\sigma_x = 40 \text{ MPa}$, $\sigma_y = -10 \text{ MPa}$, $\tau_{xy} = 0 \text{ MPa}$, $\theta = -15^\circ$

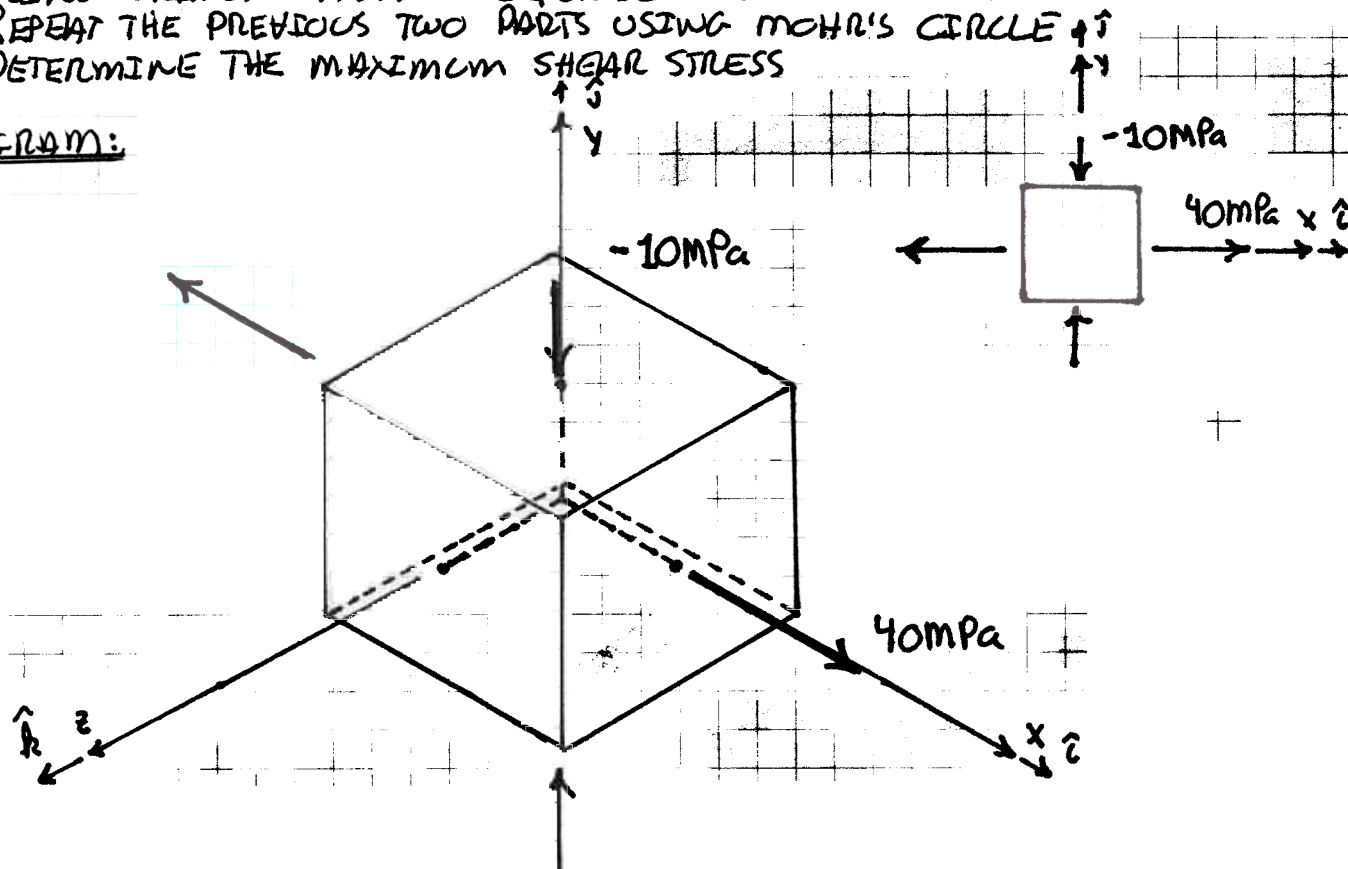
ASSUMPTIONS

1. PLANE STRESS: $\sigma_z = \tau_{xz} = \tau_{yz} = 0$

FIND:

- DRAW THE ELEMENT RELATIVE TO THE xy PLANE
- USING TRANSFORMATION EQUATIONS DETERMINE $[\sigma]$ FOR $\theta = -15^\circ$
- USING TRANSFORMATION EQUATIONS DETERMINE THE PRINCIPAL STRESSES
- REPEAT THE PREVIOUS TWO PARTS USING MOHR'S CIRCLE
- DETERMINE THE MAXIMUM SHEAR STRESS

DIAGRAM:



SOLUTION:

USING TRANSFORMATION EQUATIONS, THE STRESS ON THE ELEMENT ROTATED -15° ABOUT THE Z-AXES ARE

$$\begin{aligned}\sigma_{x'} &= \sigma_x \cos^2 \theta + \sigma_y \sin^2 \theta + 2 \tau_{xy} \cos \theta \sin \theta \\ &= (40 \text{ MPa}) \cos^2(-15^\circ) + (-10 \text{ MPa}) \sin^2(-15^\circ) + 2 \cdot (0) \cdot \cos(-15^\circ) \sin(-15^\circ) \\ &= \boxed{36.65 \text{ MPa}}\end{aligned}$$

$$\begin{aligned}\sigma_{y'} &= \sigma_x \sin^2 \theta + \sigma_y \cos^2 \theta - 2 \tau_{xy} \sin \theta \cos \theta \\ &= (40 \text{ MPa}) \sin^2(-15^\circ) + (-10 \text{ MPa}) \cos^2(-15^\circ) - 2 \cdot (0) \cdot \sin(-15^\circ) \cos(-15^\circ) \\ &= \boxed{-6.65 \text{ MPa}}\end{aligned}$$

$$\begin{aligned}\tau_{x'y'} &= -(\sigma_x - \sigma_y) \sin \theta \cos \theta + \tau_{xy} (\cos^2 \theta - \sin^2 \theta) \\ &= -(40 \text{ MPa} - (-10 \text{ MPa})) \sin(-15^\circ) \cos(-15^\circ) + 0 (\cos^2(-15^\circ) - \sin^2(-15^\circ)) \\ &= \boxed{12.5 \text{ MPa}}\end{aligned}$$

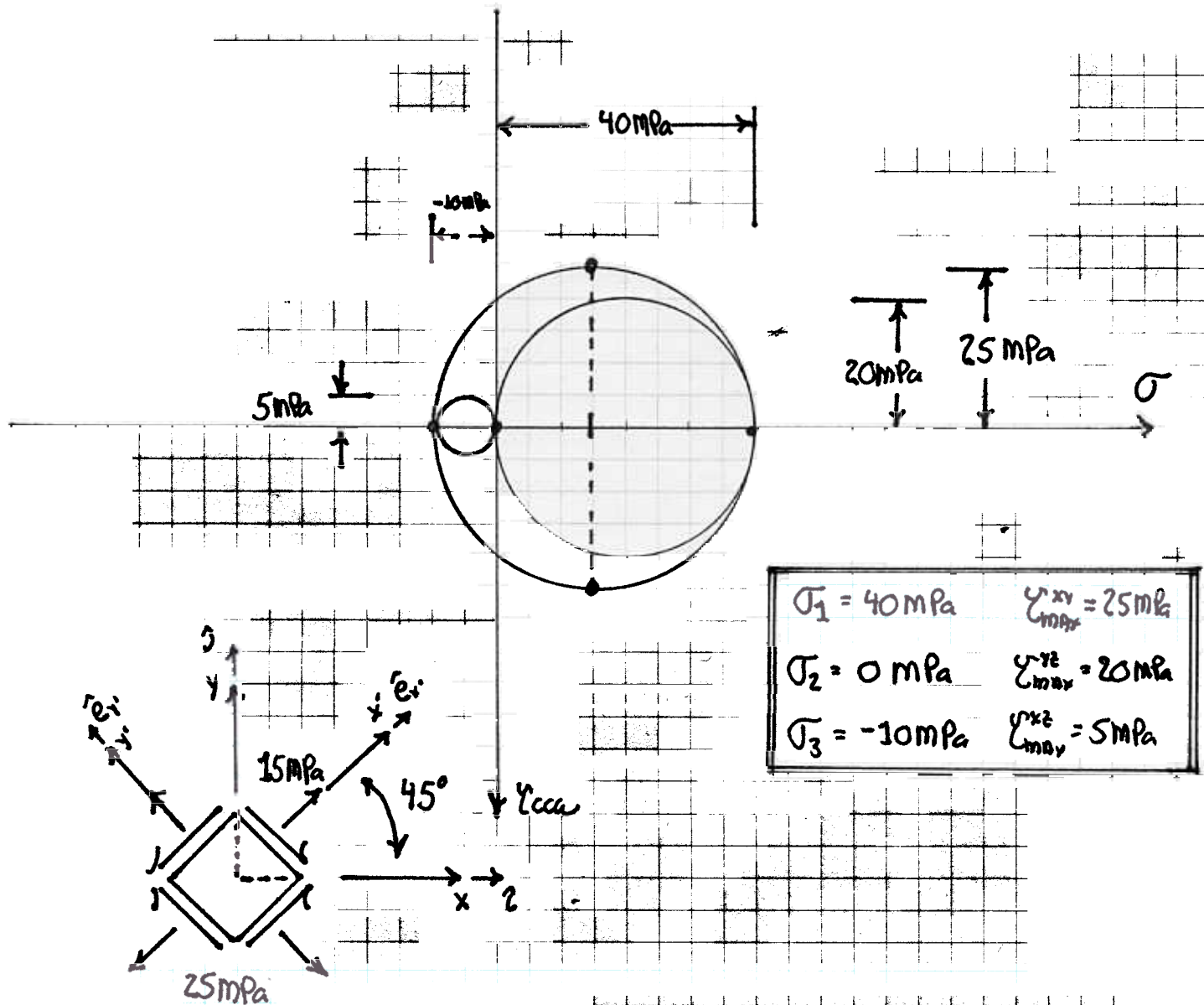
THE PRINCIPAL STRESSES, CALCULATED USING EQUATIONS ARE

$$\begin{aligned}\sigma_{1,2} &= \frac{\sigma_x + \sigma_y}{2} \pm \sqrt{\left(\frac{\sigma_x - \sigma_y}{2}\right)^2 + \tau_{xy}^2} \\ &= \frac{(40 \text{ MPa} + (-10 \text{ MPa}))}{2} \pm \sqrt{\left(\frac{(40 \text{ MPa} - (-10 \text{ MPa}))}{2}\right)^2 + (0)^2} \\ &= 15 \text{ MPa} \pm 25 \text{ MPa}\end{aligned}$$

$$\sigma_1 = 40 \text{ MPa}$$

$$\sigma_2 = -10 \text{ MPa}$$

$$\tau_{\max} = \pm \sqrt{\left(\frac{\sigma_x - \sigma_y}{2}\right)^2 + \tau_{xy}^2} = \pm \sqrt{\left(\frac{40 \text{ MPa} - (-10 \text{ MPa})}{2}\right)^2 + (0)^2} = \boxed{\pm 25 \text{ MPa}}$$



SUMMARY:

IN THIS PROBLEM THE MAXIMUM SHEAR STRESS IS IN PLANE DEFINED BY THE PLANE STRESS PROBLEM. IF THIS PROBLEM IS EXTRAPOLATED TO WHERE $\sigma_x = -\sigma_y$, THE MAXIMUM SHEAR STRESS WOULD BE ORIENTED ON AN ELEMENT WITH ZERO NORMAL STRESS.