

VP160 Recitation Class IV

Angular Momentum & Rigid Body Dynamics Part I

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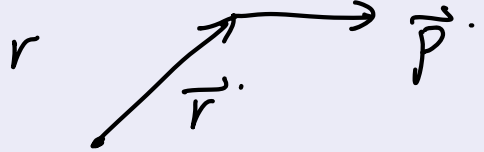
1 Angular Momentum

2 Rigid Body Dynamics Part I

Angular Momentum

$$\vec{L} = \vec{r} \times \vec{p}$$

$[kg \cdot m^2 / s]$



Angular Momentum

$$\vec{L} = \vec{r} \times \vec{p}$$

$$[kg \cdot m^2 / s]$$

How to derive?

$$\vec{F} = \frac{d\vec{p}}{dt} = m \cdot a + \left(\frac{dm}{dt} \right) \cdot v$$

Angular Momentum

$$\vec{L} = \vec{r} \times \vec{p}$$

$$[kg \cdot m^2 / s]$$

How to derive?

$$\vec{F} = \frac{d\vec{p}}{dt}$$

$$\Rightarrow \vec{r} \times \vec{F} = \vec{r} \times \frac{d\vec{p}}{dt} = \left[\frac{d}{dt} (\vec{r} \times \vec{p}) - \frac{d\vec{r}}{dt} \times \vec{p} \right]$$

Angular Momentum

$$\vec{L} = \vec{r} \times \vec{p}$$

$$[kg \cdot m^2 / s]$$

How to derive?

$$\vec{F} = \frac{d\vec{p}}{dt}$$

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Notice $\boxed{\frac{d\vec{r}}{dt} \times \vec{p}} = \vec{v} \times (m\vec{v}) = 0$

$\vec{v} \times m(\vec{v})$

Angular Momentum

$$\vec{L} = \vec{r} \times \vec{p}$$

$$[kg \cdot m^2 / s]$$

How to derive?

$$\vec{F} = \frac{d\vec{p}}{dt}$$

$$\Rightarrow \vec{r} \times \vec{F} = \vec{r} \times \frac{d\vec{p}}{dt} = \frac{d}{dt}(\vec{r} \times \vec{p}) - \frac{d\vec{r}}{dt} \times \vec{p}$$

Notice $\frac{d\vec{r}}{dt} \times \vec{p} = \vec{v} \times (m\vec{v}) = 0$

$$\begin{aligned} \mathcal{M} &= \vec{r} \times \vec{F} \\ \mathcal{M} &= \frac{d\vec{L}}{dt} \end{aligned}$$

$$\Rightarrow \underbrace{\vec{r} \times \vec{F}}_{\vec{\tau}} = \frac{d}{dt} \underbrace{(\vec{r} \times \vec{p})}_L$$

$\vec{\tau}$: torque

Angular Momentum Theorem

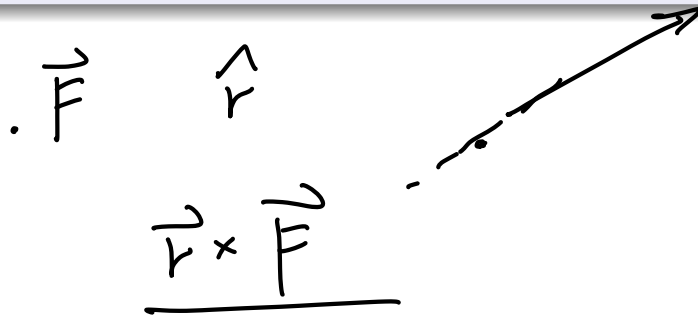
$$\vec{\tau} = \frac{d\vec{L}}{dt} \Rightarrow \underbrace{\vec{L}(t_2) - \vec{L}(t_1)} = \int_{t_1}^{t_2} \vec{\tau} dt$$

Angular Momentum Theorem

$$\vec{\tau} = \frac{d\vec{L}}{dt} \Rightarrow \boxed{\vec{L}(t_2) - \vec{L}(t_1) = \int_{t_1}^{t_2} \vec{\tau} dt}$$

Law of Conservation of Angular Momentum $L(t) = L(0)$

$$\text{If } \vec{\tau} = 0 \Rightarrow \vec{L} = \text{const}$$



Angular Momentum Theorem

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Law of Conservation of Angular Momentum

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Applications:

- Central force field ($\vec{\tau} = \vec{r} \times \vec{F} = 0$)

Angular Momentum Theorem

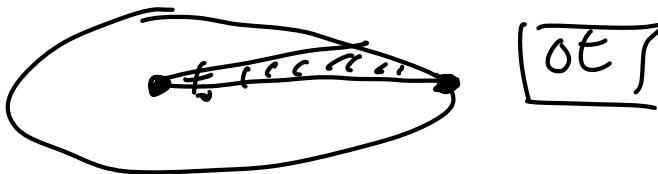
$$\vec{\tau} = \frac{d\vec{L}}{dt} \Rightarrow \vec{L}(t_2) - \vec{L}(t_1) = \int_{t_1}^{t_2} \vec{\tau} dt$$

Law of Conservation of Angular Momentum

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Applications:

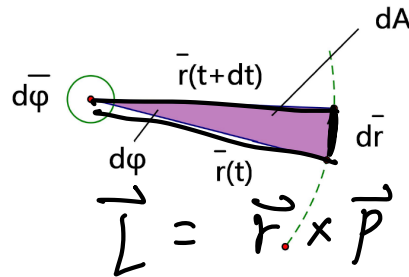
- Central force field ($\vec{\tau} = \vec{r} \times \vec{F} = 0$)
- Aerial velocity, e.g. motion of planets, Kepler's Second Laws



For planer motion, the **aerial velocity** may be defined

$$\frac{dA}{dt} = \frac{1}{2} \left| \vec{r} \times \frac{d\vec{r}}{dt} \right|$$

$$= \frac{1}{2} |\vec{r} \times \vec{v}|$$



$$\vec{L} = \text{const}$$

$$\vec{r} \times \vec{v}$$

The surface area swept by \vec{r} over the time dt is $dA = \left| \frac{1}{2} \vec{r} \times d\vec{r} \right|$ and the rate of change of that area

$$\bar{\sigma} = \frac{dA}{dt} = \frac{1}{2} \left| \vec{r} \times \frac{d\vec{r}}{dt} \right| = \frac{1}{2} |\vec{r} \times \vec{v}|.$$

Aerial velocity vector (direction — right-hand rule)

$$\bar{\sigma} = \frac{1}{2} (\vec{r} \times \vec{v}) \quad (\text{direction same as } d\varphi)$$

Recall: $\vec{L} = \vec{r} \times \vec{p} = \underbrace{\vec{r} \times m\vec{v}}_{=\bar{\sigma} \cdot 2m}$. Hence $\vec{L} = \text{const} \Leftrightarrow \bar{\sigma} = \text{const}$.

Consequently, for motion in a central force field $\bar{\sigma} = \text{const}$.

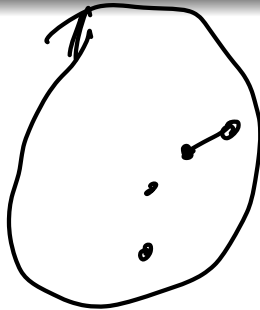
$$F = -\frac{dU}{dr}$$

Handwritten notes and diagrams on the right side of the slide, including a diagram of a central force field and a note about the direction of the force vector.

Angular Momentum in System of Particles

Conservation of the Angular Momentum Law

If the net torque of external forces on a system of particles is equal to zero, then the total angular momentum of that system is conserved.



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$$\boxed{\frac{d\vec{L}}{dt}} = \underbrace{\vec{\tau}}_{=0} = \tau_{ext} + \underbrace{\tau_{int}}_{=0} \approx 0$$

Angular Momentum in System of Particles

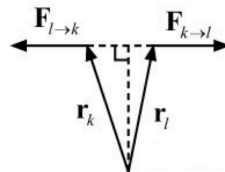
Conservation of the Angular Momentum Law

If the net torque of external forces on a system of particles is equal to zero, then the total angular momentum of that system is conserved.

Why $\tau_{int} = 0$?

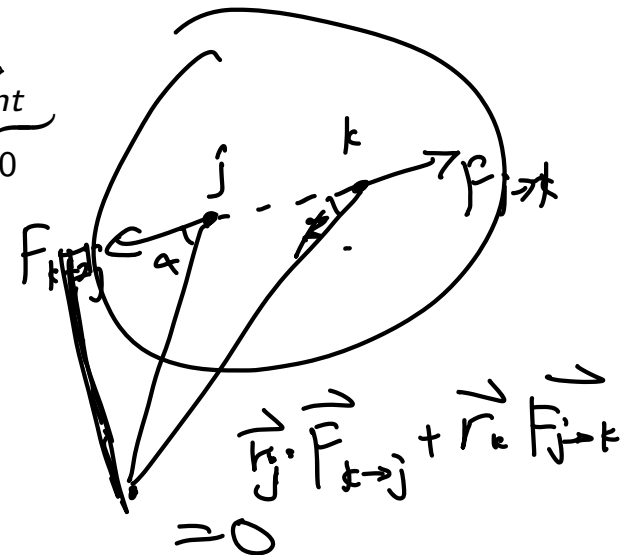
For any two particles k, l in the system,

$$\tau_{k \rightarrow l} = -\tau_{l \rightarrow k}$$



$$\frac{d\vec{L}}{dt} = \vec{\tau} = \tau_{ext} + \underbrace{\tau_{int}}_{=0}$$

$\tau \approx$



Applications:

Use with other conservation laws:

- Conservation of Energy
- Conservation of Momentum

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Exercise 1

A particle with mass m is put into a force field $\vec{F} = \alpha \vec{r}$, where α is a positive constant. The particle's initial velocity is \vec{v}_0 and its initial position is P_0 , when it moves to the position P_e , the instantaneous velocity \vec{v}_e is orthogonal to its radius vector \vec{r}_e . Take $\alpha = \frac{mv_0^2}{4a^2}$ and calculate the value of

$$\frac{|\vec{v}_e|}{|\vec{v}_0|} \cdot \vec{r} \times \vec{F} = 0.$$

$$-\frac{\partial U}{\partial r} = F(r) = \alpha r.$$

$$\Rightarrow U(r) = C - \frac{1}{2}\alpha r^2 = U(0) - \frac{1}{2}\alpha r^2$$

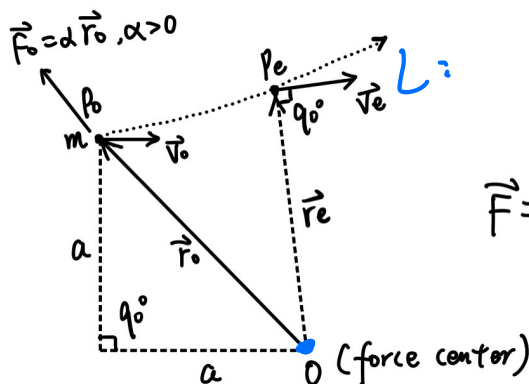
$$U(r)$$

$$E: \frac{1}{2}mv_0^2 - \frac{1}{2}\alpha r_0^2 = \frac{1}{2}mv_e^2 - \frac{1}{2}\alpha r_e^2$$

$$L: m v_0 r_0 \frac{\sqrt{2}}{2} = m v_e \cdot r_e.$$

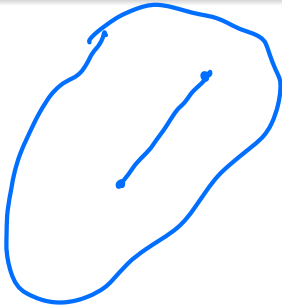
$$\vec{F} = \alpha \vec{r}$$

$$\frac{|\vec{v}_e|}{|\vec{v}_0|} = \frac{1}{2}\sqrt{1+\sqrt{5}}$$



Rigid Body

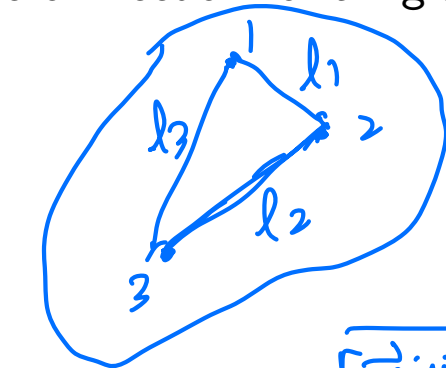
A body is called rigid if $|\vec{r}_i - \vec{r}_j| = \text{const}$ for any point i, j in the body.



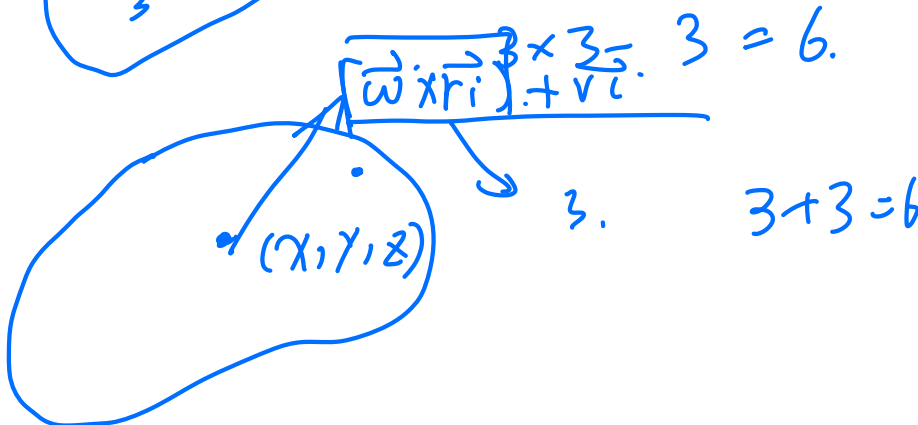
Rigid Body

A body is called rigid if $|\vec{r}_i - \vec{r}_j| = \text{const}$ for any point i, j in the body.

Degree of freedom of a rigid body?



$$\left\{ \begin{aligned} (x_1 - x_2)^2 + (y_1 - y_2)^2 + (z_1 - z_2)^2 &= l_1^2 \\ &= l_2^2 \\ &= l_3^2 \end{aligned} \right.$$



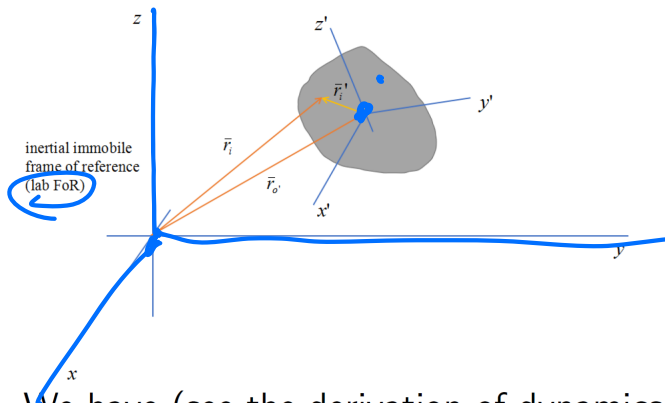
$$3 \times 3 = 6.$$

$$3. \quad 3 + 3 = 6$$

Rigid Body

A body is called rigid if $|\vec{r}_i - \vec{r}_j| = \text{const}$ for any point i, j in the body.

Degree of freedom of a rigid body?



FoR associated with the rigid body is, in general, non-inertial —the body can move arbitrarily.

O' — a point of the body

We have (see the derivation of dynamics in non-inertial FoRs)

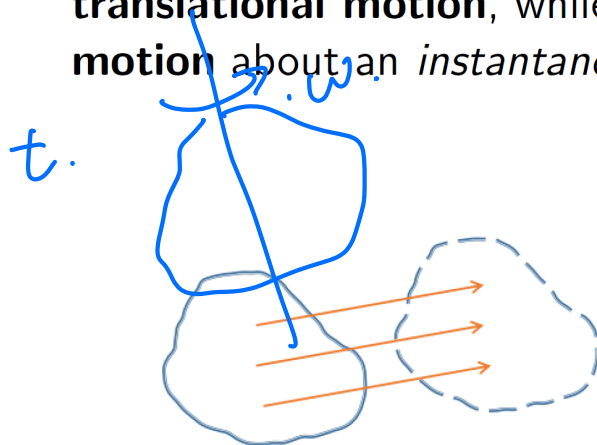
$$\begin{aligned} \vec{r}_i &= \vec{r}_{O'} + \vec{r}'_i, \\ \vec{v}_i &= \boxed{\vec{v}_{O'}} + \underbrace{\vec{v}'_i}_{=0} + \boxed{\vec{\omega} \times \vec{r}'_i}, \end{aligned} \quad \text{Ⓢ} \times \vec{r}'_i$$

where $\vec{v}'_i = 0$ due to the fact that the body is rigid (no relative motion of the rigid body's points).

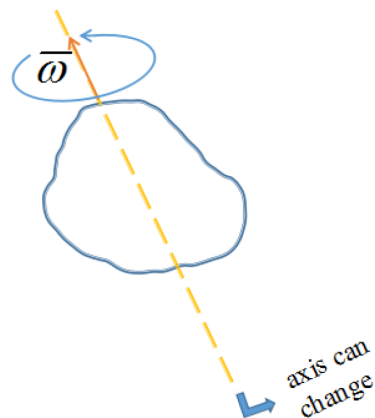
Eventually, the velocity of any point of a rigid body

$$\bar{v}_i = \bar{v}_{O'} + \bar{\omega} \times \bar{r}'_i.$$

The first term on the right hand side corresponds to the **translational motion**, while the second term to the **rotational motion** about an *instantaneous axis of rotation*.



translational motion



rotational motion

Consequently, the total momentum of an arbitrarily moving rigid body (in lab FoR) is

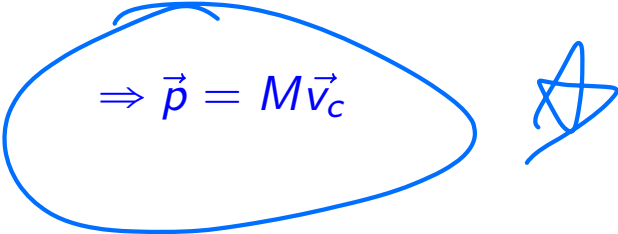
$$\begin{aligned}
 \vec{P} &= \sum_{i=1}^N m_i \vec{v}_i = \sum_{i=1}^N m_i \vec{v}_{O'} + \sum_{i=1}^N m_i (\vec{\omega} \times \vec{r}_{i'}) = \\
 &= M \vec{v}_{O'} + \vec{\omega} \times \underbrace{\sum_{i=1}^N m_i \vec{r}_{i'}}_{M \vec{r}'_{cm}} = \underbrace{M \vec{v}_{O'}}_{\text{translational motion}} + \underbrace{M \vec{\omega} \times \vec{r}'_{cm}}_{\text{rotational motion}} = 0.
 \end{aligned}$$

Handwritten notes and derivations:

- $M \vec{r}'_{cm}$ (circled)
- $M \cdot \sum_{i=1}^N \frac{m_i \vec{r}_{i'}}{M} = \sum m_i$
- $\vec{P} = M \cdot \vec{v}_{cm}$ (boxed)

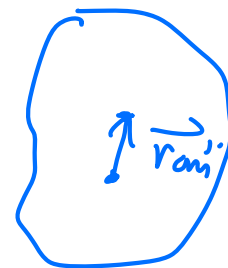
Consequently, the total momentum of an arbitrarily moving rigid body (in lab FoR) is

$$\begin{aligned}\bar{\mathbf{P}} &= \sum_{i=1}^N m_i \bar{\mathbf{v}}_i = \underbrace{\sum_{i=1}^N m_i \bar{\mathbf{v}}_{O'}}_M + \sum_{i=1}^N m_i (\bar{\boldsymbol{\omega}} \times \bar{\mathbf{r}}_{i'}) = \\ &= M \bar{\mathbf{v}}_{O'} + \bar{\boldsymbol{\omega}} \times \underbrace{\sum_{i=1}^N m_i \bar{\mathbf{r}}_{i'}}_{M \bar{\mathbf{r}}'_{\text{cm}}} = \underbrace{M \bar{\mathbf{v}}_{O'}}_{\text{translational motion}} + \underbrace{M \bar{\boldsymbol{\omega}} \times \bar{\mathbf{r}}'_{\text{cm}}}_{\text{rotational motion}}\end{aligned}$$


$$\Rightarrow \vec{p} = M \vec{v}_c$$

In the lab FoR

$$\begin{aligned}
 \bar{L} &= \sum_{i=1}^N \bar{L}_i = \sum_{i=1}^N m_i \bar{r}_i \times \bar{v}_i = \sum_{i=1}^N [m_i (\bar{r}_{O'} + \bar{r}'_i) \times (\bar{v}_{O'} + \bar{\omega} \times \bar{r}'_i)] \\
 &= \sum_{i=1}^N m_i (\bar{r}_{O'} \times \bar{v}_{O'}) + \sum_{i=1}^N m_i \bar{r}_{O'} \times (\bar{\omega} \times \bar{r}'_i) + \\
 &\quad + \sum_{i=1}^N m_i \bar{r}'_i \times \bar{v}_{O'} + \sum_{i=1}^N m_i \bar{r}'_i \times (\bar{\omega} \times \bar{r}'_i) \\
 &= M \bar{r}_{O'} \times \bar{v}_{O'} + \underbrace{[M \bar{r}_{O'} \times (\bar{\omega} \times \bar{r}'_{cm}) + M \bar{r}'_{cm} \times \bar{v}_{O'}]}_{\text{X}} + \\
 &\quad + \sum_{i=1}^N m_i \bar{r}'_i \times (\bar{\omega} \times \bar{r}'_i)
 \end{aligned}$$



In the lab FoR

$$\begin{aligned}
 \vec{L} &= \sum_{i=1}^N \vec{L}_i = \sum_{i=1}^N m_i \vec{r}_i \times \vec{v}_i = \sum_{i=1}^N [m_i (\vec{r}_{O'} + \vec{r}'_i) \times (\vec{v}_{O'} + \vec{\omega} \times \vec{r}'_i)] \\
 &= \sum_{i=1}^N m_i (\vec{r}_{O'} \times \vec{v}_{O'}) + \sum_{i=1}^N m_i \vec{r}_{O'} \times (\vec{\omega} \times \vec{r}'_i) + \\
 &\quad + \sum_{i=1}^N m_i \vec{r}'_i \times \vec{v}_{O'} + \sum_{i=1}^N m_i \vec{r}'_i \times (\vec{\omega} \times \vec{r}'_i) \\
 &= M \vec{r}_{O'} \times \vec{v}_{O'} + M \vec{r}_{O'} \times (\vec{\omega} \times \vec{r}'_{cm}) + M \vec{r}'_{cm} \times \vec{v}_{O'} + \\
 &\quad + \sum_{i=1}^N m_i \vec{r}'_i \times (\vec{\omega} \times \vec{r}'_i)
 \end{aligned}$$

$$\Rightarrow \vec{L} = \underbrace{\vec{L}_c}_{= M \vec{r}_c \times \vec{v}_c} + \vec{L}'$$

$M \vec{r}_c \times \vec{v}_c$

In the lab FoR

$$\begin{aligned}
 \bar{\mathbf{L}} &= \sum_{i=1}^N \bar{\mathbf{L}}_i = \sum_{i=1}^N m_i \bar{\mathbf{r}}_i \times \bar{\mathbf{v}}_i = \sum_{i=1}^N [m_i (\bar{\mathbf{r}}_{O'} + \bar{\mathbf{r}}'_i) \times (\bar{\mathbf{v}}_{O'} + \bar{\boldsymbol{\omega}} \times \bar{\mathbf{r}}'_i)] \\
 &= \sum_{i=1}^N m_i (\bar{\mathbf{r}}_{O'} \times \bar{\mathbf{v}}_{O'}) + \sum_{i=1}^N m_i \bar{\mathbf{r}}_{O'} \times (\bar{\boldsymbol{\omega}} \times \bar{\mathbf{r}}'_i) + \\
 &\quad + \sum_{i=1}^N m_i \bar{\mathbf{r}}'_i \times \bar{\mathbf{v}}_{O'} + \sum_{i=1}^N m_i \bar{\mathbf{r}}'_i \times (\bar{\boldsymbol{\omega}} \times \bar{\mathbf{r}}'_i) \\
 &= M \bar{\mathbf{r}}_{O'} \times \bar{\mathbf{v}}_{O'} + M \bar{\mathbf{r}}_{O'} \times (\bar{\boldsymbol{\omega}} \times \bar{\mathbf{r}}'_{\text{cm}}) + M \bar{\mathbf{r}}'_{\text{cm}} \times \bar{\mathbf{v}}_{O'} + \\
 &\quad + \sum_{i=1}^N m_i \bar{\mathbf{r}}'_i \times (\bar{\boldsymbol{\omega}} \times \bar{\mathbf{r}}'_i)
 \end{aligned}$$

$$\Rightarrow \vec{\mathbf{L}} = \underbrace{\vec{\mathbf{L}}_c}_{=M\vec{\mathbf{r}}_c \times \vec{\mathbf{v}}_c} + \vec{\mathbf{L}}'$$

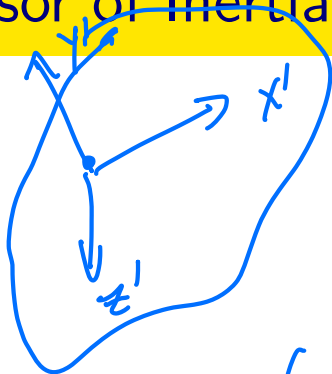
$\vec{\mathbf{L}}' = \sum_{i=1}^N m_i \bar{\mathbf{r}}'_i \times (\bar{\boldsymbol{\omega}} \times \bar{\mathbf{r}}'_i)$: Rigid body's angular momentum w.r.t its center of mass

Tensor of Inertia

$$\vec{L}' = \sum_{i=1}^N m_i \vec{r}'_i \times (\vec{\omega} \times \vec{r}'_i) = \boxed{I \vec{\omega}}$$

I

Tensor of Inertia



$$\vec{L} = I \vec{\omega}$$

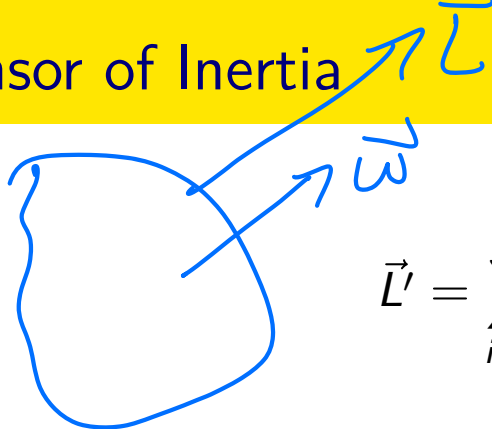
$$\vec{L}' = \sum_{i=1}^N m_i \vec{r}'_i \times (\vec{\omega} \times \vec{r}'_i) = I \vec{\omega}$$

$$\begin{pmatrix} L_{x'} \\ L_{y'} \\ L_{z'} \end{pmatrix} = \begin{bmatrix} I_{x'x'} & I_{x'y'} & I_{x'z'} \\ I_{y'x'} & I_{y'y'} & I_{y'z'} \\ I_{z'x'} & I_{z'y'} & I_{z'z'} \end{bmatrix} \begin{pmatrix} \omega_x \\ \omega_y \\ \omega_z \end{pmatrix}$$

$$\begin{bmatrix} \sum_{i=1}^N m_i (y_i'^2 + z_i'^2) & \sum_{i=1}^N -m_i x'_i y'_i & \sum_{i=1}^N -m_i x'_i z'_i \\ \sum_{i=1}^N -m_i y'_i x'_i & \sum_{i=1}^N m_i (x_i'^2 + z_i'^2) & \sum_{i=1}^N -m_i y'_i z'_i \\ \sum_{i=1}^N -m_i z'_i x'_i & \sum_{i=1}^N -m_i z'_i y'_i & \sum_{i=1}^N m_i (x_i'^2 + y_i'^2) \end{bmatrix}$$

$$L_{x'} = I_{x'x'} \omega_{x'} + I_{x'y'} \omega_{y'} + I_{x'z'} \omega_{z'}$$

Tensor of Inertia



$$\vec{L}' = \sum_{i=1}^N m_i \vec{r}'_i \times (\vec{\omega} \times \vec{r}'_i) = I \vec{\omega}$$

$$I = \begin{bmatrix} I_{x'x'} & I_{x'y'} & I_{x'z'} \\ I_{y'x'} & I_{y'y'} & I_{y'z'} \\ I_{z'x'} & I_{z'y'} & I_{z'z'} \end{bmatrix} =$$

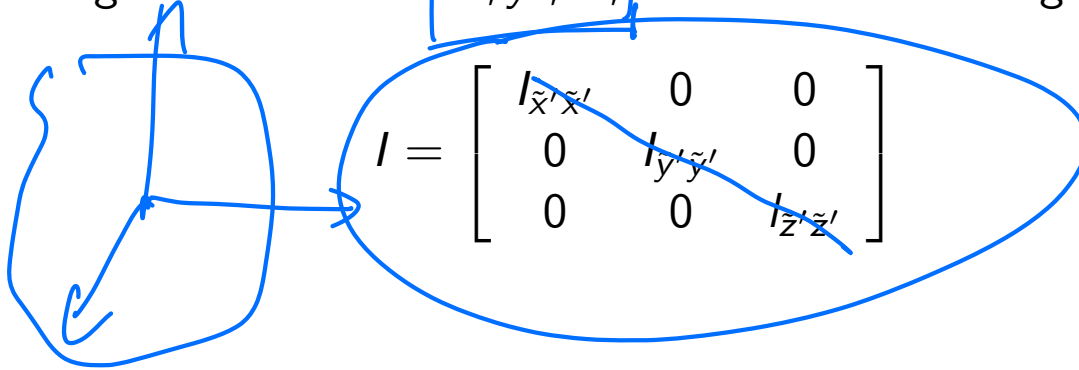
$$\begin{pmatrix} I_{x'} \\ I_{y'} \\ I_{z'} \end{pmatrix} = \lambda \begin{pmatrix} \omega_{x'} \\ \omega_{y'} \\ \omega_{z'} \end{pmatrix}$$

$$\begin{bmatrix} \sum_{i=1}^N m_i (y_i'^2 + z_i'^2) & \sum_{i=1}^N -m_i x_i' y_i' & \sum_{i=1}^N -m_i x_i' z_i' \\ \sum_{i=1}^N -m_i y_i' x_i' & \sum_{i=1}^N m_i (x_i'^2 + z_i'^2) & \sum_{i=1}^N -m_i y_i' z_i' \\ \sum_{i=1}^N -m_i z_i' x_i' & \sum_{i=1}^N -m_i z_i' y_i' & \sum_{i=1}^N m_i (x_i'^2 + y_i'^2) \end{bmatrix}$$

\vec{L} can not be always parallel to $\vec{\omega}$, when will it be?

By a random choice of x', y', z' , e.g. $\omega'_x, \omega'_y, \omega'_z$ all contributes to $L'_{x'}$, it's hard to see, but if we ...

By choosing a better set of $\tilde{x}', \tilde{y}', \tilde{z}'$, we can obtain a diagonal form of I .


$$I = \begin{bmatrix} I_{\tilde{x}'\tilde{x}'} & 0 & 0 \\ 0 & I_{\tilde{y}'\tilde{y}'} & 0 \\ 0 & 0 & I_{\tilde{z}'\tilde{z}'} \end{bmatrix}$$

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$$I = \begin{bmatrix} I_{\tilde{x}'\tilde{x}'} & 0 & 0 \\ 0 & I_{\tilde{y}'\tilde{y}'} & 0 \\ 0 & 0 & I_{\tilde{z}'\tilde{z}'} \end{bmatrix}$$

Then, ω'_x **only** contributes to \vec{L}'_x , so do ω'_y and ω'_z

$$\Rightarrow \underline{L_{x'}} = \boxed{I_{\tilde{x}'\tilde{x}'}} \omega_{x'}, \quad \underline{L_{y'}} = I_{\tilde{y}'\tilde{y}'} \cdot \omega_{y'}, \quad \underline{L_{z'}} = I_{\tilde{z}'\tilde{z}'} \cdot \omega_{z'}$$

Handwritten notes: L_x with an arrow pointing to x' and ω ; a boxed equation $\vec{L} = \boxed{I} \vec{\omega}$.

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$$I = \begin{bmatrix} I_{\tilde{x}'\tilde{x}'} & 0 & 0 \\ 0 & I_{\tilde{y}'\tilde{y}'} & 0 \\ 0 & 0 & I_{\tilde{z}'\tilde{z}'} \end{bmatrix}$$

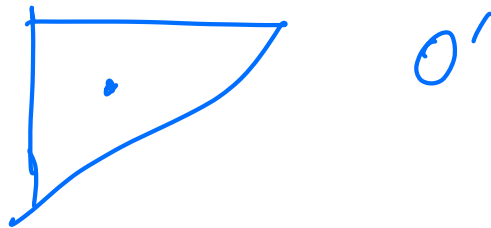
Then, ω'_x only contributes to \vec{L}'_x , so do ω'_y and ω'_z

$$\Rightarrow L_{x'} = I_{\tilde{x}'\tilde{x}'} \cdot \omega_{x'}, \quad L_{y'} = I_{\tilde{y}'\tilde{y}'} \cdot \omega_{y'}, \quad L_{z'} = I_{\tilde{z}'\tilde{z}'} \cdot \omega_{z'}$$

The axis in this special sets of axes is called the **Principal axis**, which is our main focus.

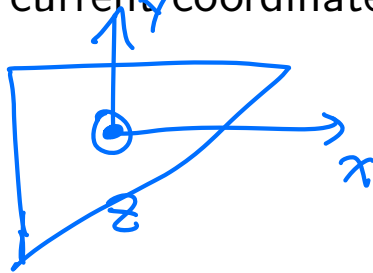
How to find principle axis

- 1 Find the mass center C of the rigid body, let C be the origin



How to find principle axis

- 1 Find the mass center C of the rigid body, let C be the origin
- 2 Use the current x, y, z coordinates to derive the tensor of inertia I



How to find principle axis

- ① Find the mass center C of the rigid body, let C be the origin
- ② Use the current coordinates x, y, z to derive the tensor of inertia I
- ③ Let $\det(I - \lambda \mathbb{1}) = 0$ to find $\lambda_1, \lambda_2, \lambda_3$.

Handwritten notes illustrating the derivation of the inertia tensor I and the characteristic equation for principal axes.

The inertia tensor I is represented as a matrix:

$$I = \begin{pmatrix} I_{xx} - \lambda & & \\ & I_{yy} - \lambda & \\ & & \dots \end{pmatrix}$$

The determinant of the matrix $I - \lambda \mathbb{1}$ is set to zero to find the principal axes:

$$\det \begin{pmatrix} x_1 & x_2 & x_3 \\ y_1 & y_2 & y_3 \\ z_1 & z_2 & z_3 \end{pmatrix} = 0$$

The determinant is expanded using cofactor expansion along the first row:

$$x_1 \begin{vmatrix} y_2 & y_3 \\ z_2 & z_3 \end{vmatrix} - x_2 \begin{vmatrix} y_1 & y_3 \\ z_1 & z_3 \end{vmatrix} + x_3 \begin{vmatrix} y_1 & y_2 \\ z_1 & z_2 \end{vmatrix} = 0$$

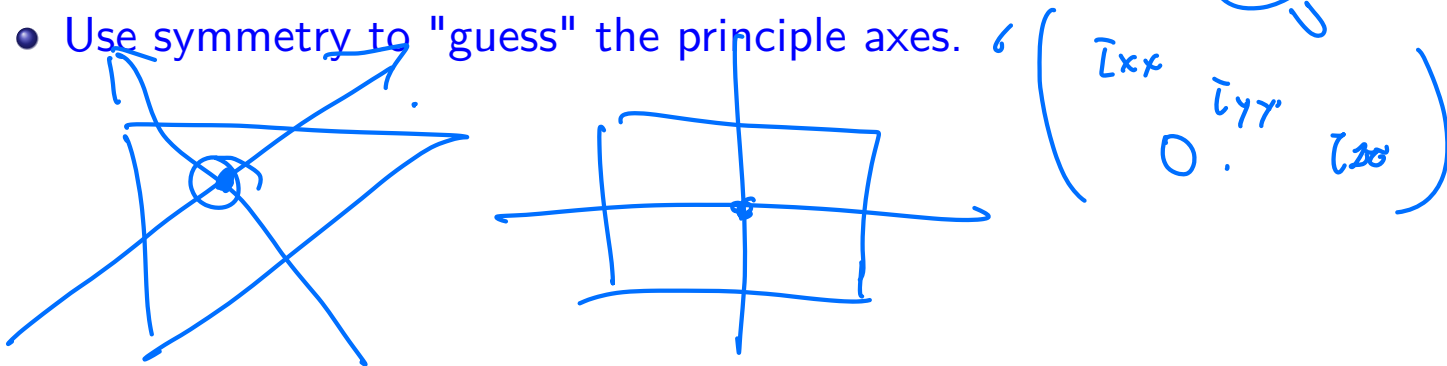
How to find principle axis

- ① Find the mass center C of the rigid body, let C be the origin
- ② Use the current coordinates x, y, z to derive the tensor of inertia I
- ③ Let $\det(I - \lambda \mathbb{1}) = 0$ to find $\lambda_1, \lambda_2, \lambda_3$.
- ④ Plug back λ_i into the equation $(I - \lambda_i \mathbb{1})\vec{u}_i = 0$, find the solution $\vec{u}_1, \vec{u}_2, \vec{u}_3$

$$(I - \lambda \mathbb{1}) \begin{bmatrix} \vec{u}_i \end{bmatrix} = 0.$$

How to find principle axis

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- ⑤ Use the direction of $\vec{u}_1, \vec{u}_2, \vec{u}_3$ as axes, calculate the new I_p .
- Use symmetry to "guess" the principle axes.



How to find principle axis

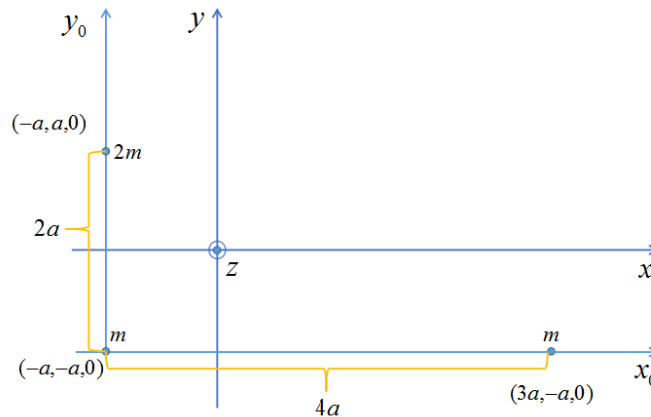
- ① Find the mass center C of the rigid body, let C be the origin
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 - Why does this methods always works?

How to find principle axis

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 - ⑤ Use the direction of $\vec{u}_1, \vec{u}_2, \vec{u}_3$ as axes, calculate the new I_p .
- Use symmetry to "guess" the principle axes.
 - Why does this methods always works?
 - Recall the form of I , it's a self-adjoint matrix.
- To learn the mathematical details, have a look at:
zxj_Eigenvalue & Diagonalization.pdf (under canvas RC folder).

Exercise 2

Use the example in slide(s-21hp14) to practice.



(answer: in slide)

Reference



Yigao Fang.

VP160 Recitation Slides.
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Haoyang Zhang.

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