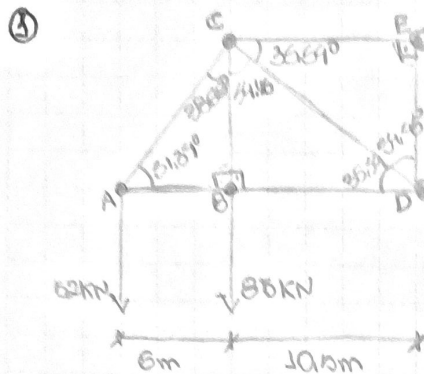


# MECÂNICA DOS SÓLIDOS 3

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$$A = 1600 \text{ mm}^2$$

$$E = 800 \text{ GPa}$$

• Adicionando uma carga  $P$  vertical em A, temos:

$$\sum F_x = P - E_x + D_x = 0$$

$$\sum F_y = E_y - 52 - 85 = 0$$

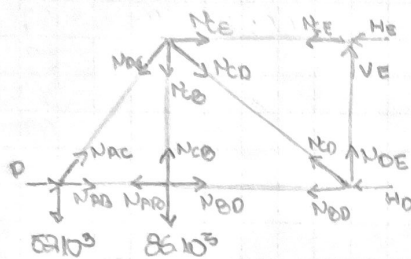
$$\sum M_E = 52 \cdot 16.5 + 85 \cdot 10.5 + D_x \cdot 7.5 + 7.5P = 0$$

$$E_y = 137 \text{ kN}$$

$$D_x = P - 233.4 \text{ kN}$$

$$E_x = 233.4 \text{ kN}$$

• Método dos nós



$$AC = 9.6 \text{ m}; CD = 12.9 \text{ m}$$

$$\sum F_H = -H_D - N_{BD} - N_{CD} \cos(35.46) = 0$$

$$\sum F_V = N_{DE} + N_{CD} \cos(54.46) = 0$$

$$\sum F_H = N_{BD} - N_{AB} = 0$$

$$\sum F_V = N_{CB} - 85 \cdot 10^3 = 0$$

$$\sum F_H = N_{AB} + N_{AC} \cos(51.34) + P = 0$$

$$\sum F_V = N_{AE} - 52 \cdot 10^3 - 52 \cdot 10^3 = 0$$

$$\sum F_H = N_{CE} - N_{AC} \cos(38.66) + N_{CD} \cos(35.46) = 0$$

$$\sum F_V = -N_{AC} \cos(38.66) - N_{CB} - N_{CD} \cos(54.46) = 0$$

$$\sum F_H = -H_E - N_{CE} = 0$$

$$\sum F_V = V_E - N_{DE} = 0$$

• Para barras:

$$\Delta = \sum_{i=1}^n \int_0^{L_i} \frac{N_i}{E \cdot A_i} \cdot \frac{\partial N_i}{\partial P} dx$$

$$\Delta_{AB} = \int_0^{10.5} \frac{(1 - 43623.75 - P) \cdot (-1)}{EA} dx$$

$$\Delta_{AB} = 0.0008 \text{ m}$$

$$\Delta_{AC} = \int_0^{9.6} \frac{66.598 \cdot 0}{EA} dx$$

$$\Delta_{AC} = 0$$

$$\Delta_{BC} = \int_0^{7.5} \frac{85 \cdot 0}{EA} dx$$

$$\Delta_{BC} = 0$$

$$\Delta_{BD} = \int_0^{10.5} \frac{(-43623.75 - P) \cdot (-1)}{EA} dx$$

$$\Delta_{BD} = 0.0014 \text{ m}$$

$$\Delta_{CD} = \int_0^{12.9} \frac{-235.7 \cdot 0}{EA} dx$$

$$\Delta_{CD} = 0$$

$$\Delta_{CE} = \int_0^{12.9} \frac{233.395 \cdot 0}{EA} dx$$

$$\Delta_{CE} = 0$$

$$\Delta_{DE} = \int_0^{7.5} \frac{137000 \cdot 0}{EA} dx$$

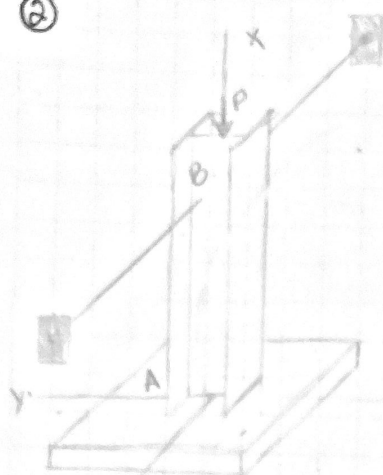
$$\Delta_{DE} = 0$$

$$\Delta_A = \Delta_{AB} + \Delta_{AC} + \Delta_{BC} + \Delta_{BD} + \Delta_{CD} + \Delta_{CE} + \Delta_{DE}$$

$$\Delta_A = 0.0022 \text{ m}$$

Logo o deslocamento em B será de 2,2 mm para a direita

②



$$\sigma_{adm} = 250 \text{ MPa}$$

$$E = 200 \text{ GPa}$$

$$I_x = 128 \cdot 10^6 \text{ mm}^4$$

$$I_y = 18,4 \cdot 10^6 \text{ mm}^4$$

$$h_x = 130 \text{ mm}$$

Coef de segurança de flambagem 2

9m

- Considerando a flambagem em torno do eixo  $x_1$ :

$$P_{cr} = \frac{\pi^2 E I_x}{(KL)^2} = \frac{\pi^2 (200 \text{ N/mm}^2) (128 \cdot 10^6 \text{ mm}^4)}{[2,9 \text{ m} \cdot 1 \text{ mm/m}]^2} = 779,82 \text{ kN}$$

- Em torno do eixo  $y$ :

$$P_{cr} = \frac{\pi^2 E I_y}{(KL)^2} = \frac{\pi^2 (200 \text{ N/mm}^2) (18,4 \cdot 10^6 \text{ mm}^4)}{[0,7 \cdot 9 \text{ m} \cdot 1 \text{ mm/m}]^2} = 935,1 \text{ kN}$$

Logo, o  $P_{cr}$  adotado será 779,82 kN

- Para a tensão crítica:

$$\sigma_{cr} = \frac{\pi^2 E}{(KL/r)^2} = \frac{\pi^2 (200 \text{ N/mm}^2)}{(2,9 \text{ m} \cdot 1 \text{ mm/m} / 130 \text{ mm})^2} = 102,9 \text{ MPa}$$

Como  $\sigma_{cr} < \sigma_{adm}$ , confirmamos a escolha de  $P_{cr}$  adotado

- Para critério de estabilidade:

$$\sigma \leq \frac{\sigma_{cr}}{n_p} ; \frac{P}{A} \leq \frac{P_{cr}}{A_{1p}} ; P \leq \frac{779,82}{2} ; P \leq 389,91 \text{ kN}$$

Carga admissível: 389,91 kN