

CONFIDENTIAL
THE UNIVERSITY OF SYDNEY

TABLE NUMBER

FAMILY NAME

GIVEN NAMES

STUDENT NUMBER

FACULTY OF ENGINEERING

AMME3500

Systems Dynamics and Control

Semester 1, 2009

Time allowed: 3 hours

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No copy of this paper is to be removed from the examination room by candidates or supervisors, nor may any portion of the paper be copied. All copies must be returned to the examiner.

There are six (6) questions in this exam. All questions count towards your final mark. These questions are to be completed in the **ANSWER BOOK PROVIDED**. Plotting pages for Questions 3, 4 & 6b are provided in this exam booklet and are to be returned with the answer book and exam booklet at the end of the exam period. Clearly label which question you are answering.

The total marks for this exam is **100**. This exam is an extra exam to assess your understanding of the material covered by this Unit of Study. Be sure to read through the exam in its entirety before starting to work on the solutions.

This is a closed book exam.

This exam booklet is to be handed back with your answer book.

Non-Programmable calculators are allowed.

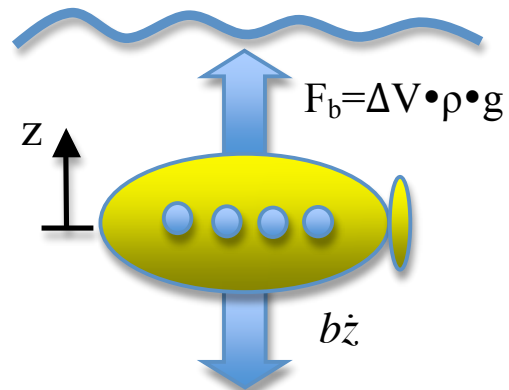
Section A – System Specifications [30 marks]

1. An LTI system has a transfer function given by:

$$T(s) = \frac{15(s + 5.2)}{s(s + 5)(s^2 + 2s + 5)(s^2 + 14s + 50)}$$

[10 marks]

- What is the system type?
 - What is the steady state error for this system?
 - You are asked to make an estimate of the percentage overshoot for the step response of the system, and you do not have access to simulation software. What is your estimate? Provide a short sentence or two of explanation as part of your answer.
2. A submarine depth control system can be achieved by varying the buoyancy of the vessel. The vehicle is usually trimmed to be neutrally buoyant such that the weight of water displaced by the volume of the vessel (the buoyant force) is offset by the weight of the vehicle. Adjusting the volume around this setpoint by pumping a non-compressible fluid into or out of a bladder will increase or decrease the buoyant force accordingly, allowing the vehicle to control its depth. The vessel will be subject to damping induced by the water, modelled here as viscous damping proportional to velocity through the water. Assume that $m=1000$ kg, $b=100$ Nsec/m and that the density of the water is $\rho=1000$ kg/m³. Based on this system, answer the following questions



[20 marks]

- Sketch a block diagram showing the relationship between the changing volume and the depth of the submersible. Find the transfer function between the volume ΔV and the depth z of the vessel.
- An engineer decides to apply a simple proportional control law to regulate the depth of the submersible such that the volume is computed so that z tracks a reference depth z_r according to the feedback law

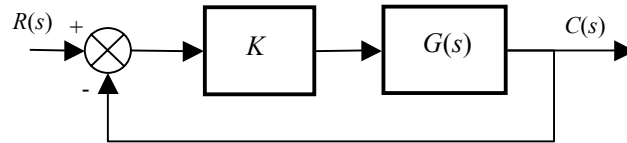
$$\Delta V = K(z_r - z)$$

Sketch a block diagram of the resulting system. Find the transfer function between z_r and z .

- What is the value of K that can be used if we wish to have a rise time of 30s for a commanded change in depth?
- What is the maximum gain K that can be used such that we have an overshoot $M_p < 5\%$ for a step change in desired position z_r ?

Section B – Root Locus and Bode plots [40 marks]

3. For a given system shown here



the transfer function between is given by

$$G(s) = \frac{s + 4}{s(s + 2)(s^2 + 8s + 32)}$$

For this system, sketch the Root Locus. Show the asymptotes and the departure/arrival angles where appropriate. Produce your sketch on the s-plane sheet at the back of this exam booklet (pg. 7) and be sure to turn in your copy of the exam booklet with your answer book at the end of the exam. Also answer the following questions (give a brief explanation) [15 marks]

- Is the system stable for small values of K?
- Is the system stable for very large values of K?

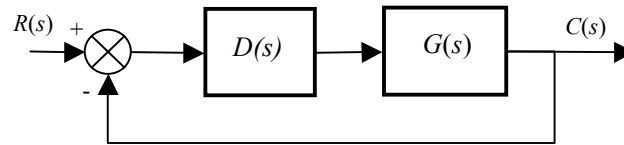
4. Based on the system $G(s)$ shown below, answer the following questions [25 marks]

$$G(s) = \frac{50(s + 4)}{s(s + 2)(s^2 + 110s + 1000)}$$

- Sketch the Bode plot for this system on the magnitude/phase diagram at the end of this exam booklet (pg. 8). Show any derivation you require in the answer book and include the asymptotes on the sketch itself.
- From your Bode plot sketched in a), estimate the 0dB frequency, Gain Margin and Phase Margin and -3dB bandwidth.
- Based on the Gain Margin found in b), determine the value of K for which the system is marginally stable.

Section C – Controller Design [30 marks]

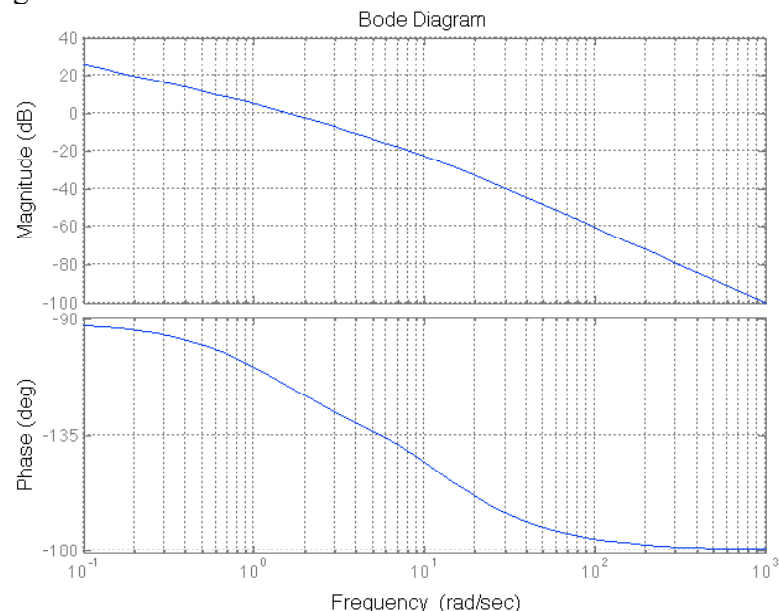
5. Given the following second order system, design a lead compensator $D(s)$ to yield a 5% overshoot with a twofold reduction in settling time. Due to noise considerations, the lead pole should be placed at $s=-15$ [15 marks]



where

$$G(s) = \frac{1}{(s+1)(s+5)}$$

6. In order to demonstrate your understanding of the controllers we have studied in this course, provide short answers to the following questions. Each response should take no more than half a page in your answer book. [15 marks]
- For a PID controller, describe the effect that the proportional, integral and differential terms have on the transient response of a system. Use control terminology, sketches of step responses and any equations to help demonstrate your understanding of these controllers.
 - Given the Bode plot shown below, sketch the response of this system to the two inputs shown at the back of this booklet (pg. 9). Draw the response overlain on the input signals. Be sure to label your diagram showing important features such as the response magnitude and phase lag/lead.



- Name two advantages of frequency response techniques over the root locus (for control system analysis and design).
- As we have seen, classical techniques are suitable for modelling and control design for Linear Time Invariant (LTI) systems. Briefly describe two system characteristics that are not easy to model using the LTI methods concentrated on in this course.

THERE ARE NO MORE QUESTIONS

Selected Equations

Time Response (First Order Systems)

$$G(s) = C_{\infty} \frac{1}{s + \sigma} \qquad t_r = \frac{2.2}{\sigma}$$

Time Response (Second Order Systems)

$$G(s) = C_{\infty} \frac{\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2} \qquad \sigma = \zeta\omega_n$$

$$t_r \cong \frac{1.8}{\omega_n} \qquad \omega_d = \omega_n \sqrt{1 - \zeta^2}$$

$$\zeta = \frac{-\ln(M_p)}{\sqrt{\pi^2 + \ln^2(M_p)}} \qquad t_s \cong \frac{4}{\sigma}$$

$$M_p = e^{-\frac{\pi\zeta}{\sqrt{1-\zeta^2}}}, 0 \leq \zeta < 1$$

$$M_p \cong \begin{cases} 5\%, \zeta = 0.7 \\ 16\%, \zeta = 0.5 \\ 20\%, \zeta = 0.45 \end{cases}$$

Steady-State Error

$$e_{step}(\infty) = \frac{1}{1 + \lim_{s \rightarrow 0} G(s)} \qquad e_{ramp}(\infty) = \frac{1}{\lim_{s \rightarrow 0} sG(s)} \qquad e_{parabola}(\infty) = \frac{1}{\lim_{s \rightarrow 0} s^2 G(s)}$$

Root Locus

$$|KG(s)H(s)| = 1 \qquad \sigma_a = \frac{\sum \text{finite poles} - \sum \text{finite zeros}}{n - m}$$

$$\angle KG(s)H(s) = (2k+1)180^\circ \qquad \theta_a = \frac{(2k+1)\pi}{n - m}$$

Frequency Response

$$M = |G(j\omega)| = |G(s)|_{s=j\omega} = \sqrt{(\operatorname{Re}[G(j\omega)])^2 + (\operatorname{Im}[G(j\omega)])^2}$$

$$\phi = \angle G(j\omega) = \tan^{-1} \left[\frac{\operatorname{Im}[G(j\omega)]}{\operatorname{Re}[G(j\omega)]} \right]$$

$$GM = 20 \log K \qquad PM = \tan^{-1} \frac{2\zeta}{\sqrt{-2\zeta^2 + \sqrt{1 + 4\zeta^4}}}$$

Laplace Transform Tables

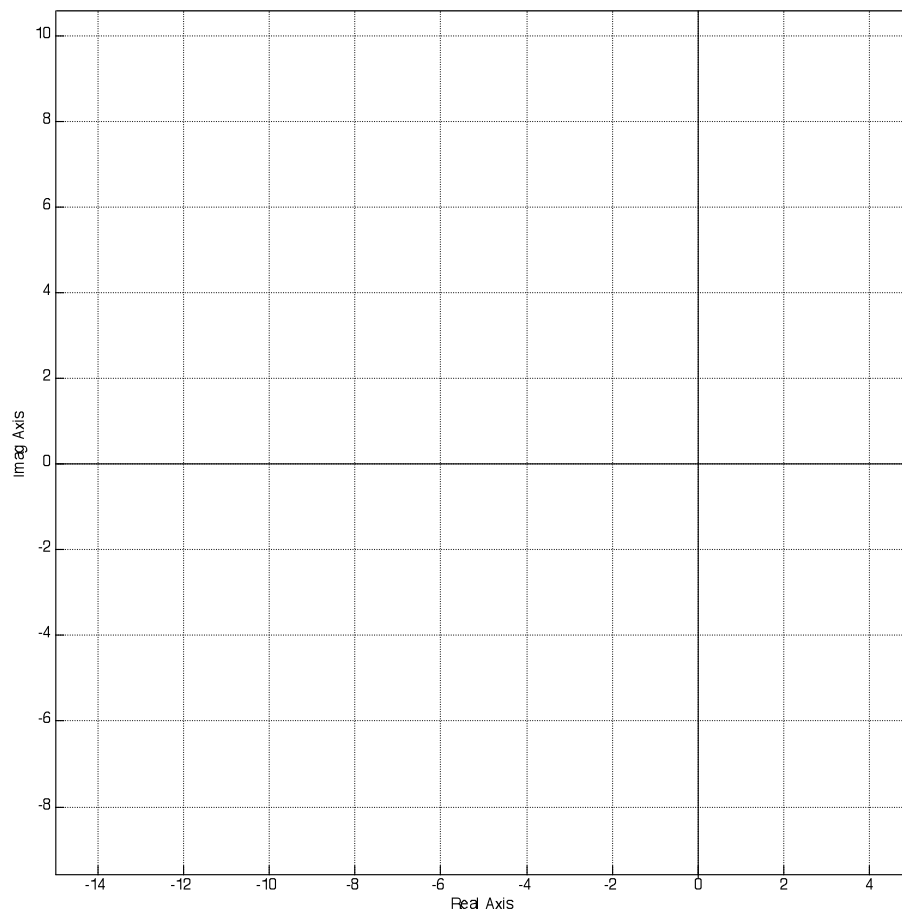
Item no.	$f(t)$	$F(s)$
1.	$\delta(t)$	1
2.	$u(t)$	$\frac{1}{s}$
3.	$tu(t)$	$\frac{1}{s^2}$
4.	$t^n u(t)$	$\frac{n!}{s^{n+1}}$
5.	$e^{-at}u(t)$	$\frac{1}{s+a}$
6.	$\sin \omega t u(t)$	$\frac{\omega}{s^2 + \omega^2}$
7.	$\cos \omega t u(t)$	$\frac{s}{s^2 + \omega^2}$

Item no.	Theorem	Name
1.	$\mathcal{L}[f(t)] = F(s) = \int_{0-}^{\infty} f(t)e^{-st} dt$	Definition
2.	$\mathcal{L}[kf(t)] = kF(s)$	Linearity theorem
3.	$\mathcal{L}[f_1(t) + f_2(t)] = F_1(s) + F_2(s)$	Linearity theorem
4.	$\mathcal{L}[e^{-at}f(t)] = F(s+a)$	Frequency shift theorem
5.	$\mathcal{L}[f(t-T)] = e^{-sT}F(s)$	Time shift theorem
6.	$\mathcal{L}[f(at)] = \frac{1}{a}F\left(\frac{s}{a}\right)$	Scaling theorem
7.	$\mathcal{L}\left[\frac{df}{dt}\right] = sF(s) - f(0-)$	Differentiation theorem
8.	$\mathcal{L}\left[\frac{d^2f}{dt^2}\right] = s^2F(s) - sf(0-) - \dot{f}(0-)$	Differentiation theorem
9.	$\mathcal{L}\left[\frac{d^nf}{dt^n}\right] = s^nF(s) - \sum_{k=1}^n s^{n-k}f^{(k-1)}(0-)$	Differentiation theorem
10.	$\mathcal{L}\left[\int_{0-}^t f(\tau) d\tau\right] = \frac{F(s)}{s}$	Integration theorem
11.	$f(\infty) = \lim_{s \rightarrow 0} sF(s)$	Final value theorem ¹
12.	$f(0+) = \lim_{s \rightarrow \infty} sF(s)$	Initial value theorem ²

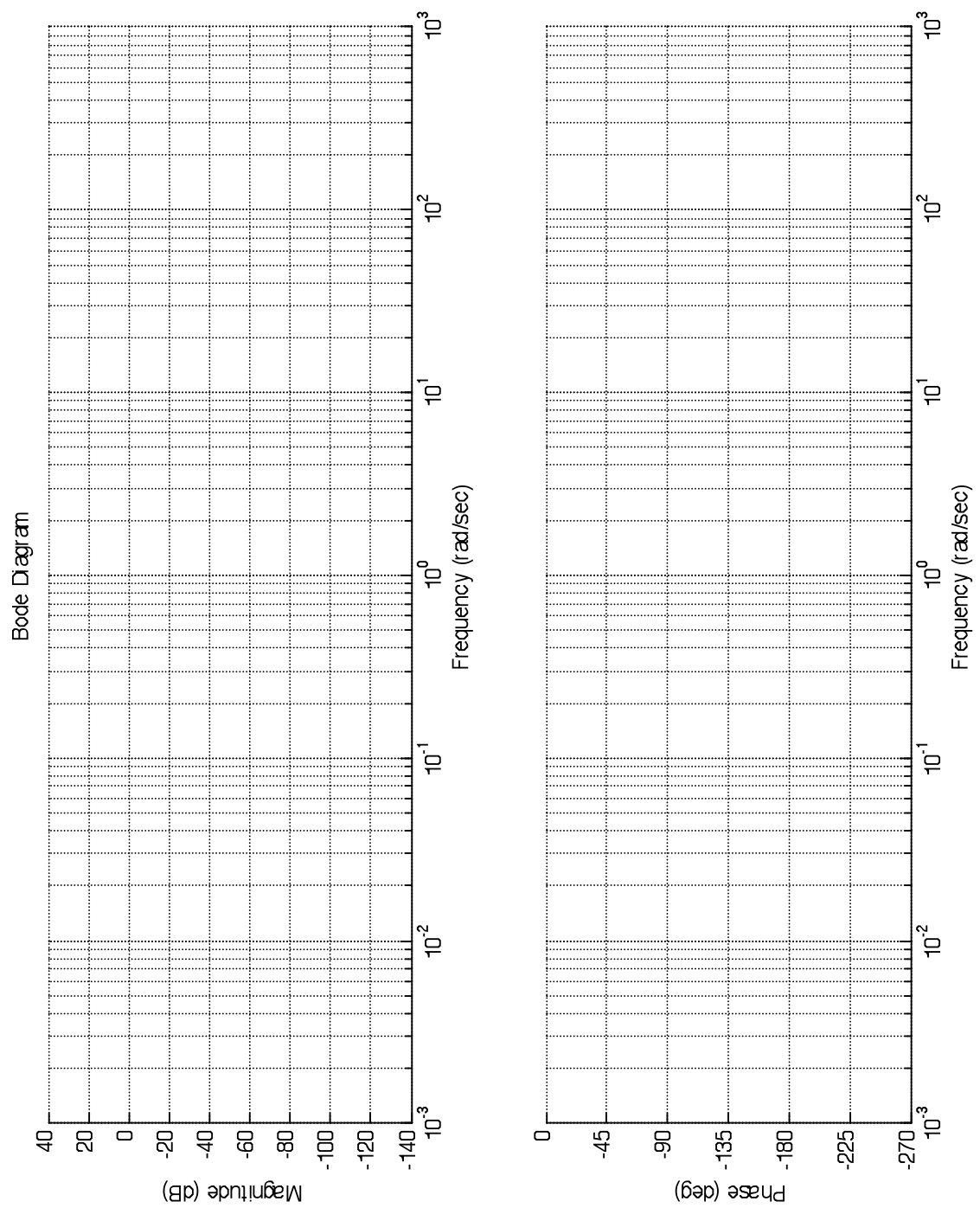
¹ For this theorem to yield correct finite results, all roots of the denominator of $F(s)$ must have negative real parts and no more than one can be at the origin.

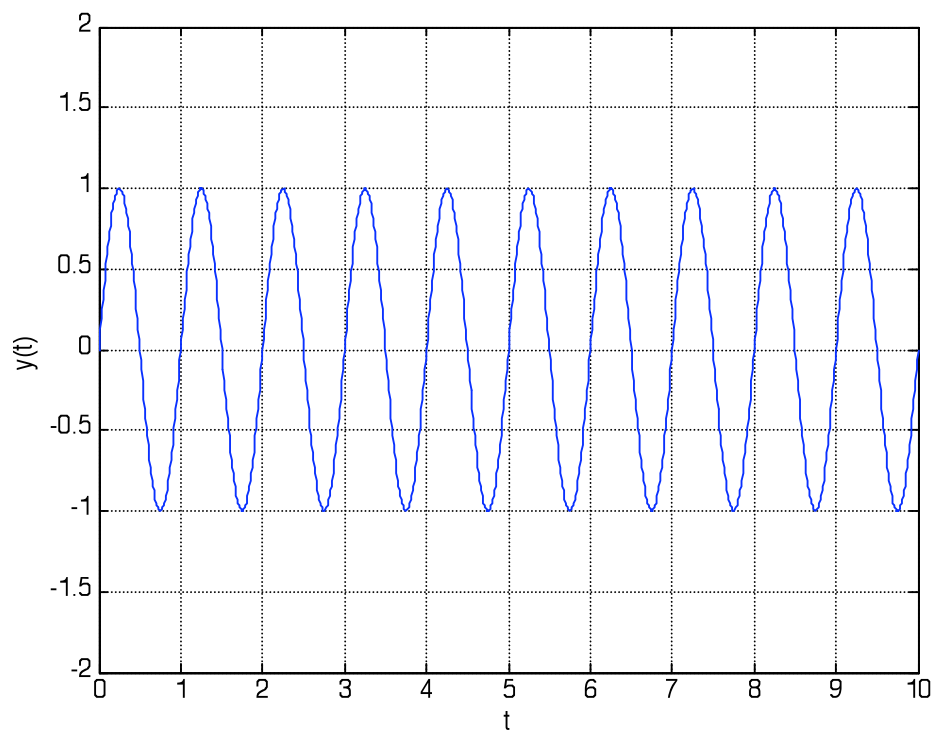
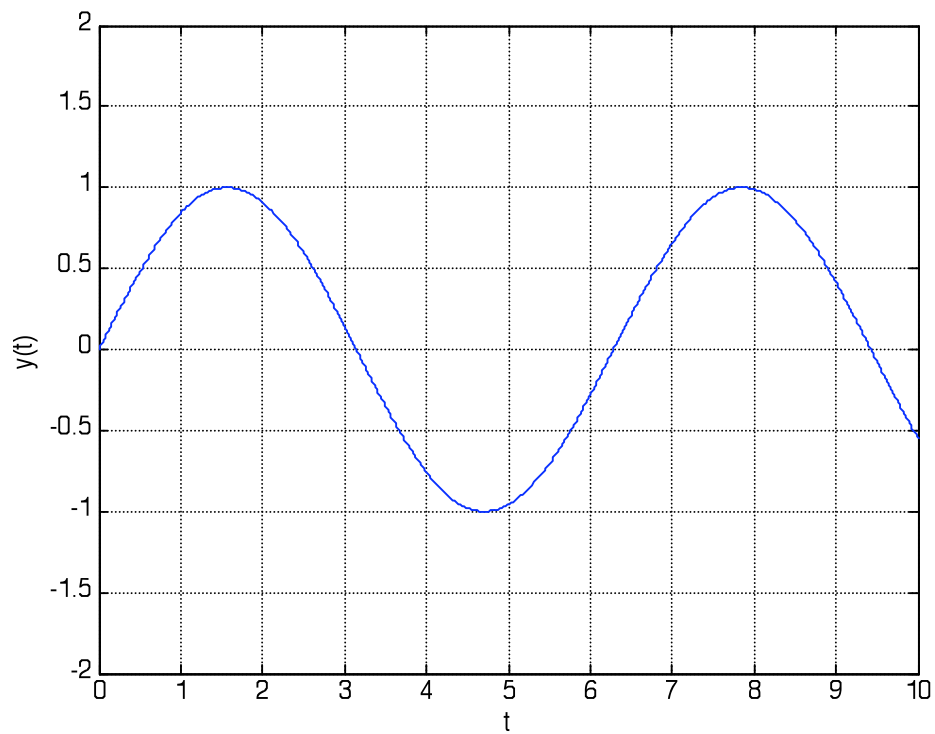
² For this theorem to be valid, $f(t)$ must be continuous or have a step discontinuity at $t = 0$ (i.e., no impulses or their derivatives at $t = 0$).

Root Locus for Question3



Bode Plot for Question 4



Input plots for Question 6b)

THIS IS THE LAST PAGE

RETURN THE EXAM WITH THE ANSWER BOOK