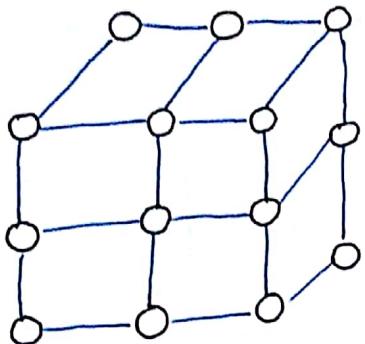


Crystal Structure



Unit cell
(smallest structure)

3 D Structure



Space-lattice

a, b, c
 α, β, γ

Type of unit cell

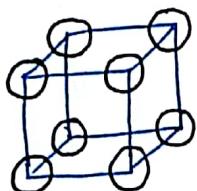
- 1. Triclinic
- 2. Mono-clinic
- 3. Orthorhombic
- 4. Rhombohedral
- 5. Tetragonal
- 6. Hexagonal
- 7. Cubic

Crystal Systems

Crystal structure \rightarrow No. of atoms present in a particular crystal system

Cubic [$a=b=c$; $\alpha=\beta=\gamma=90^\circ$]

1. simple cubic

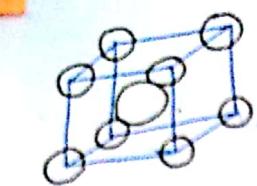


$$E.N.A. \quad \frac{1}{8} \times 8 = 1$$

$$\text{Packing Factor} = \frac{\text{Vol. of atoms}}{\text{Vol. of unit cell}}$$

$$= \frac{\frac{4}{3} \pi r_a^3}{a^3}$$

Body-centred cubic (bcc)



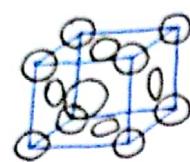
$$\text{E.N.A.} = \frac{1}{8} \times 8 + 1 = 2$$

$$4r_a = d \quad \sqrt{3}a = d$$

$$\eta = \frac{2 \times \frac{4}{3} \pi r_a^3}{a^3} =$$

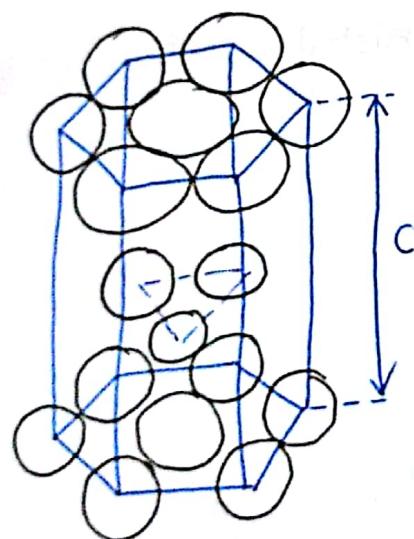
Face-centered cubic (fcc)

$$\text{E.N.A.} = 8 \times \frac{1}{8} + \frac{1}{2} \times 6 = 4$$



$$4r_a = \sqrt{2}a$$

$$\eta = \frac{4 \times \frac{4}{3} \pi r_a^3}{a^3} =$$



$$\text{E.N.A.} = 1 \text{ corner atom} + 2 \text{ face atoms} + 3 \text{ internal atoms:}$$

Al → FCC
Cu → FCC

Fe ↕ BCC
↓↑
FCC

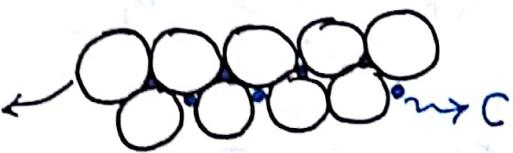
Ti → H

Alloy = Solid Solution

Interstitial type
(C-Fe alloy)

→ diameters
are very diff.
in size.

$$r_c \ll r_{Fe}$$



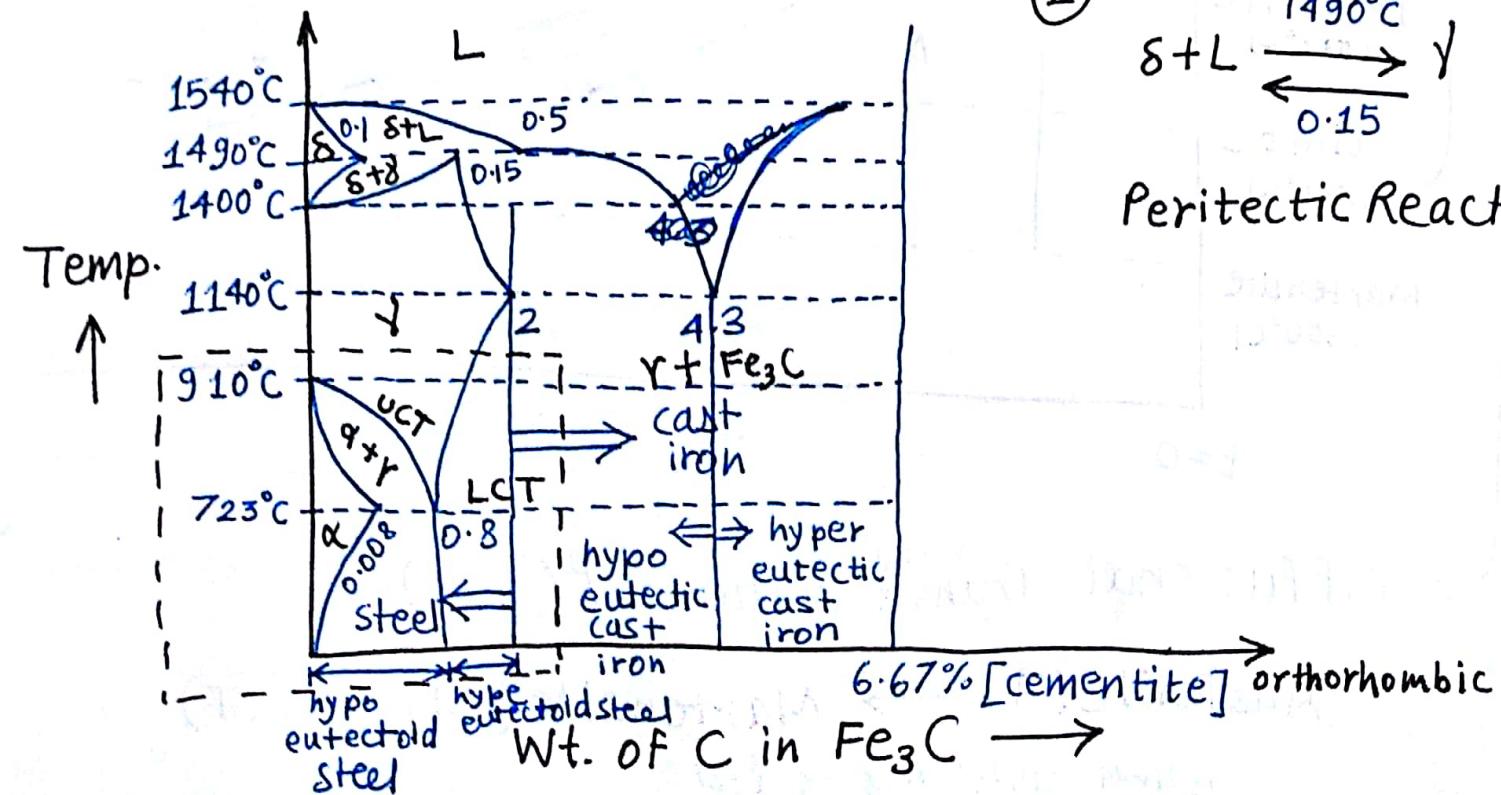
Substitute
type
(Cu-Zn alloy)

→ diameters are roughly of the same size

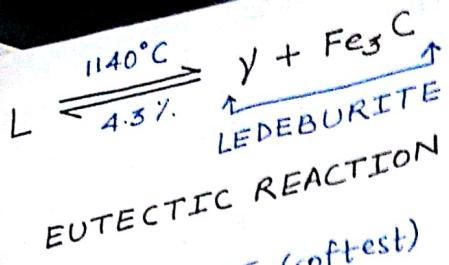
$$r_{Cu} \approx r_{Zn}$$



Diagram of Fe-Fe₃C:



③ $\gamma \rightarrow \alpha$ 0.81



EUTECTOID

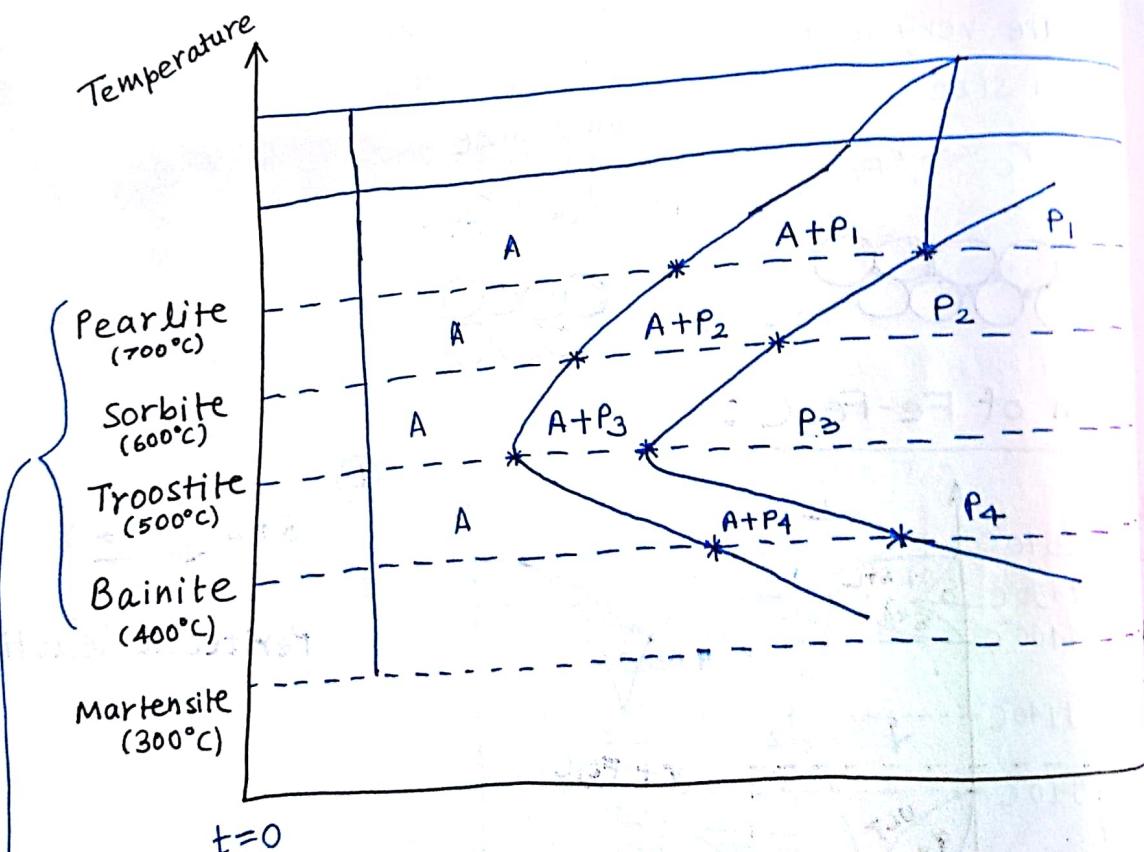
(bcc) $\alpha \rightarrow$ FERRITE (softest)

(fcc) $\gamma \rightarrow$ AUSTENITE

- 3.
- Low Carbon = up to 0.3%
 - Medium Carbon = 0.3 - 0.6%
 - High Carbon = 0.6 - 1%
 - Tool Steel = 1 - 2%

TEMPERATURE, TIME, TRANSFORMATION DIAG.

Casting

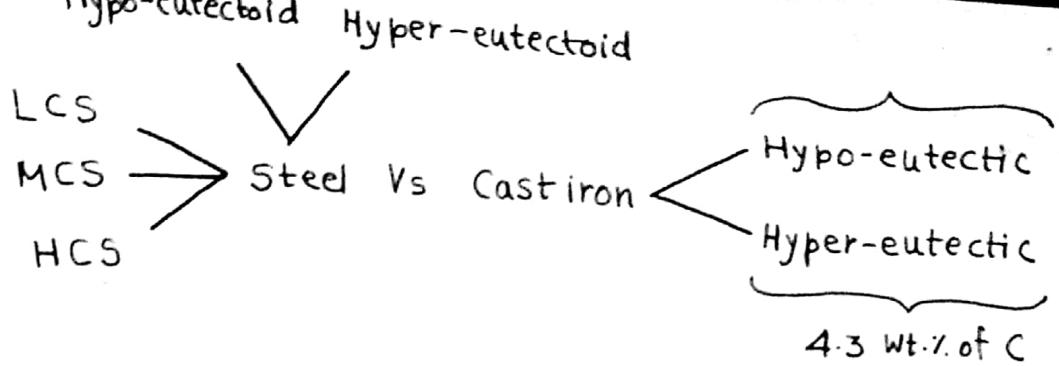


Diffusional Transformation $f(T, t)$

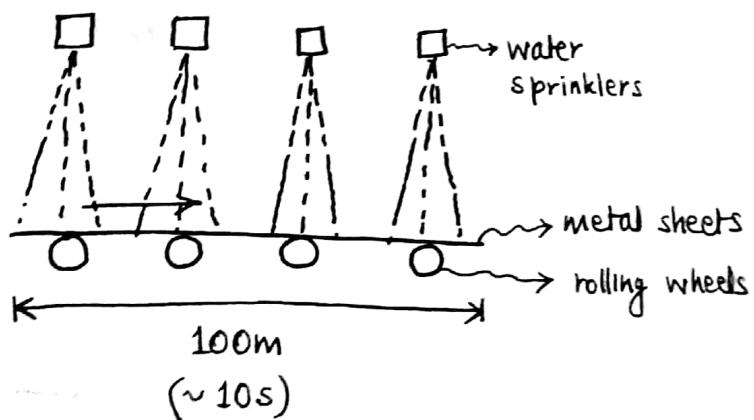
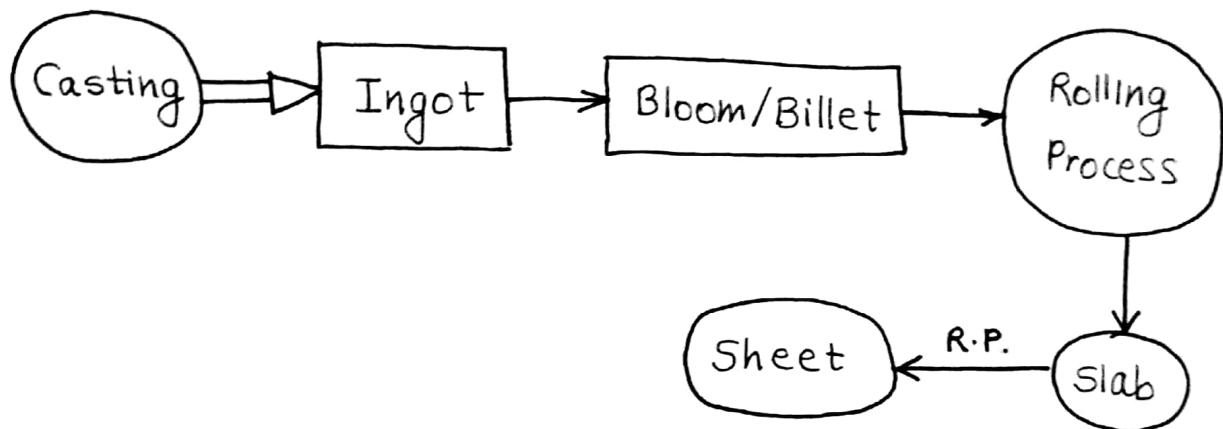
Austenite (A) \rightarrow Martensite (M) $f(T)$
when suddenly cooled

Heat Treat

- Reduce
- Refine
- Hard

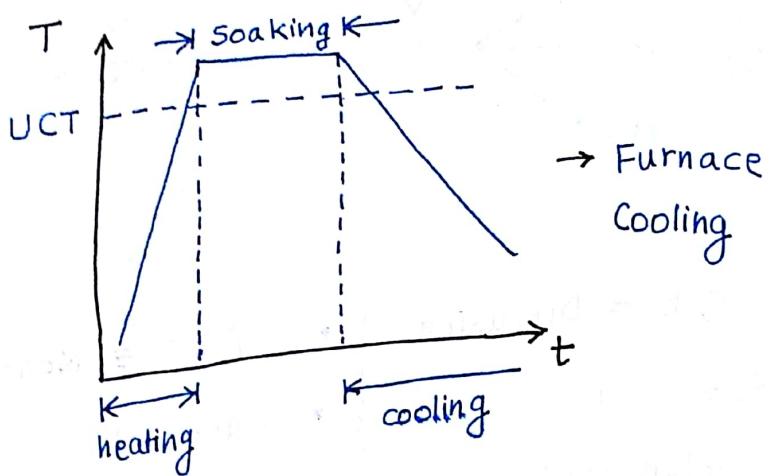


- $f(T, t) \equiv \text{Diffusion}$
 - P, S, T, B
 - complete conversion is possible
- {
- $f(T) \equiv \text{Non-diffusion}$
 - Martensite
 - complete conversion not possible



Heat Treatment

- Reduce Internal Stress
- Refine Grain Structure
- Hardness, Toughness, Ductility



2. Normalizing

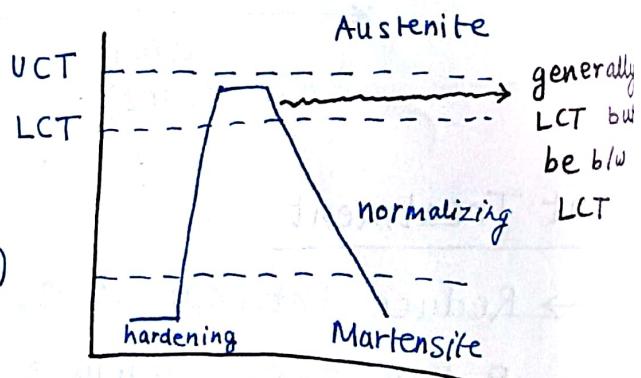
- Air Cooling
- $(\text{Cooling rate})_{\text{Normalizing}} > (\text{Cooling rate})_{\text{Annealing}}$
- $(\text{Hardness})_{\text{Normalizing}} > (\text{Hardness})_{\text{Annealing}}$

3. Hardening

- Water cooling / oil cooling
- essentially conversion of Austenite into Martensite
- very hard

4. Tempering

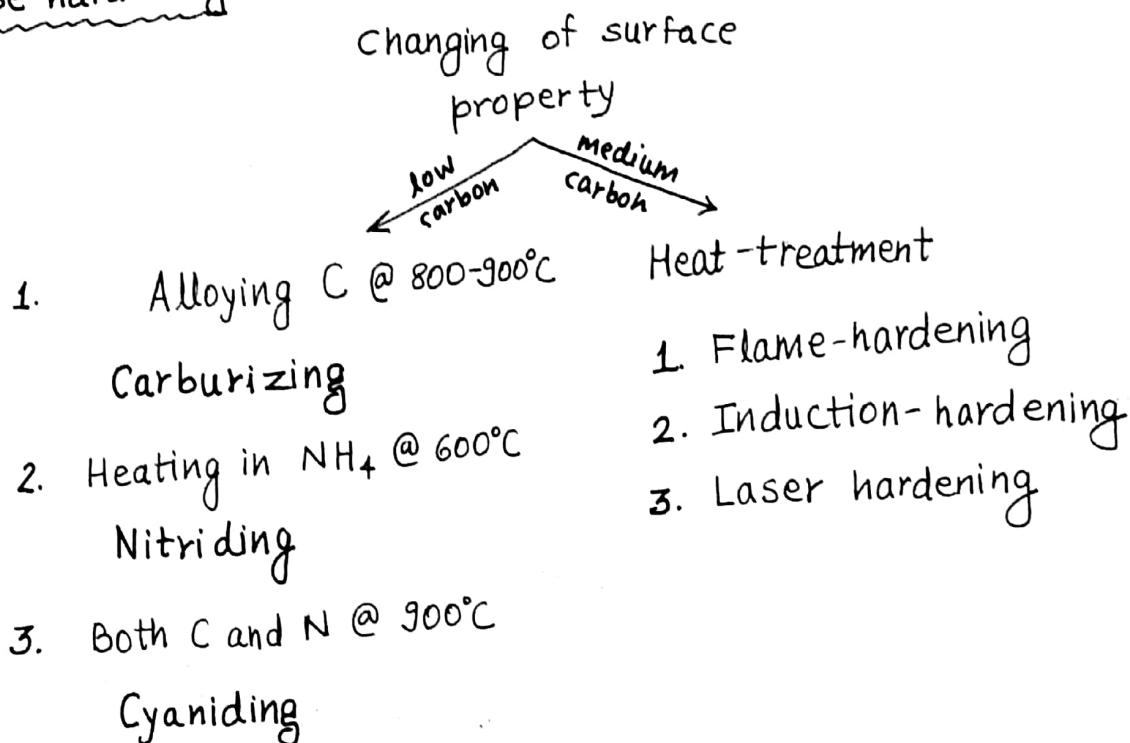
- Hardness + Ductility
- normalizing
- $(\text{Hard, strength}) + (\text{ductile} + \text{impact strength})$



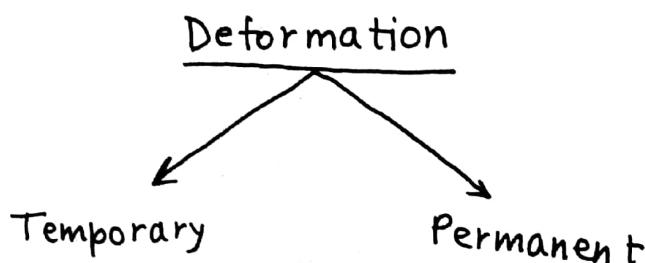
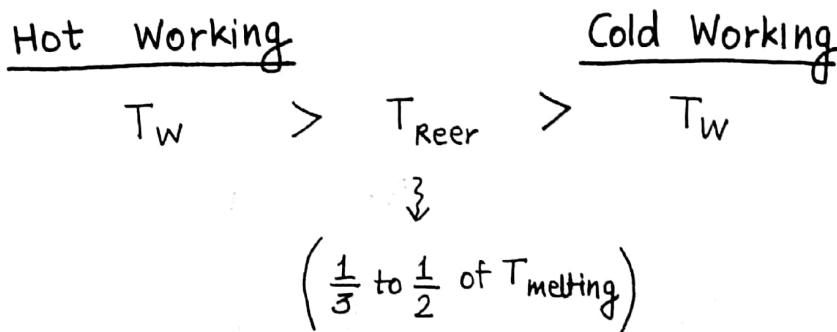
* Above 4 processes are both surface and core hardening processes. But when only surface needs to be hardened

like gears (where wear-and-tear at the surface is needed and soft core for vibration absorption), we use case hardening.

Case hardening

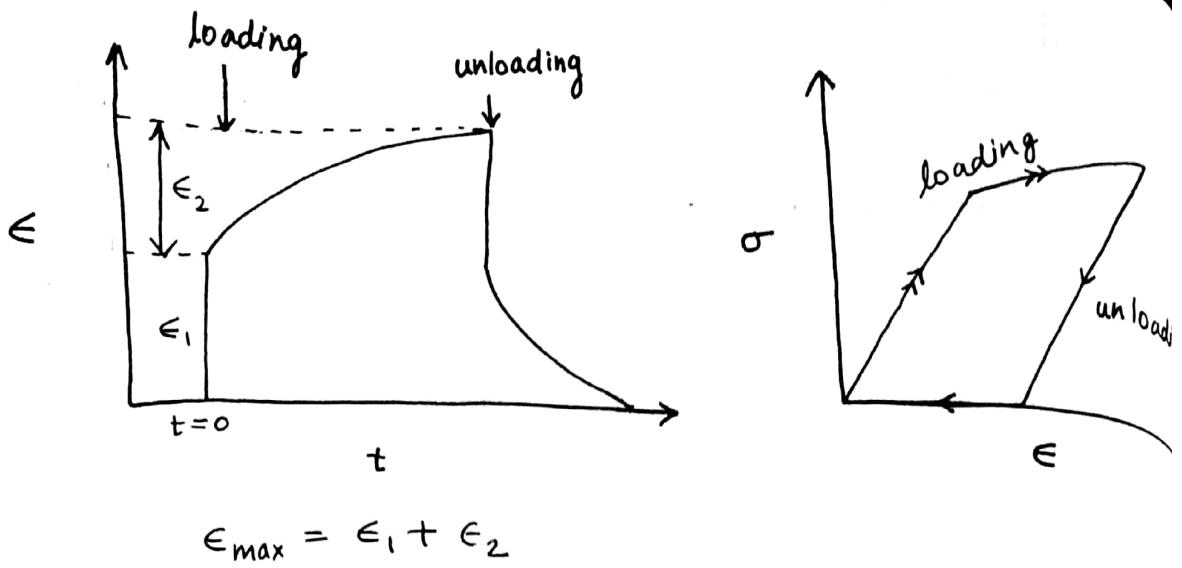


Classification of Metal Working:

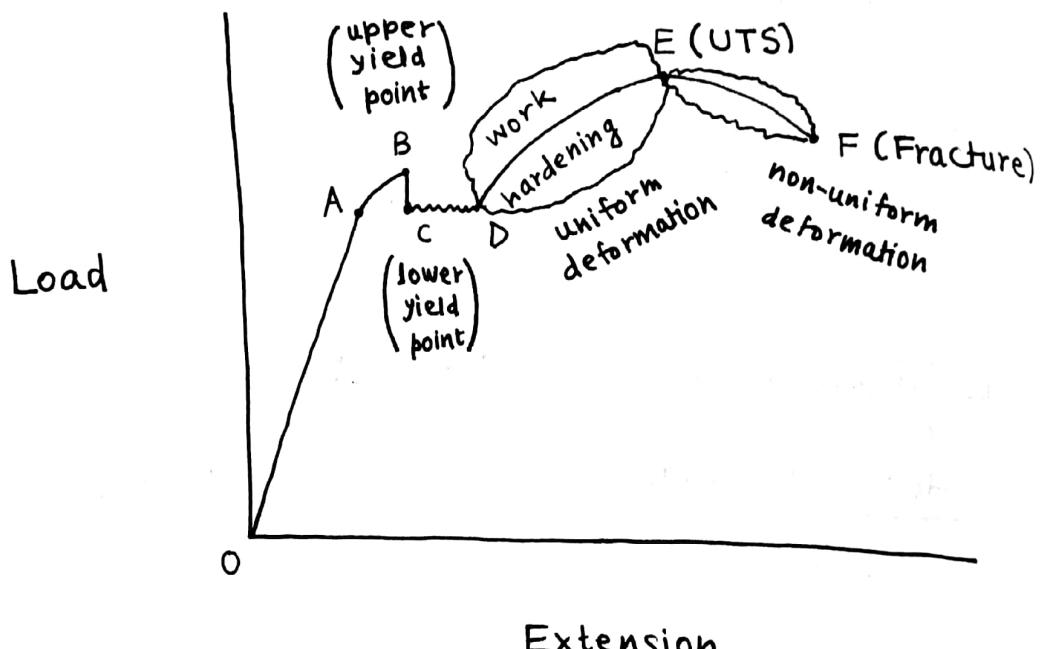


$$\text{Hooke's Law} \leftarrow \epsilon = f(\sigma)$$

$$\text{An elastic deformation} \leftarrow \epsilon = f(\sigma, t)$$



Mild Steel



$$e = \frac{\Delta L}{L_0} = \frac{L - L_0}{L_0}$$

$$S = \frac{P}{A_0}$$

$$\epsilon = \int_{l_0}^l \frac{dl}{l} = \ln\left(\frac{l}{l_0}\right)$$

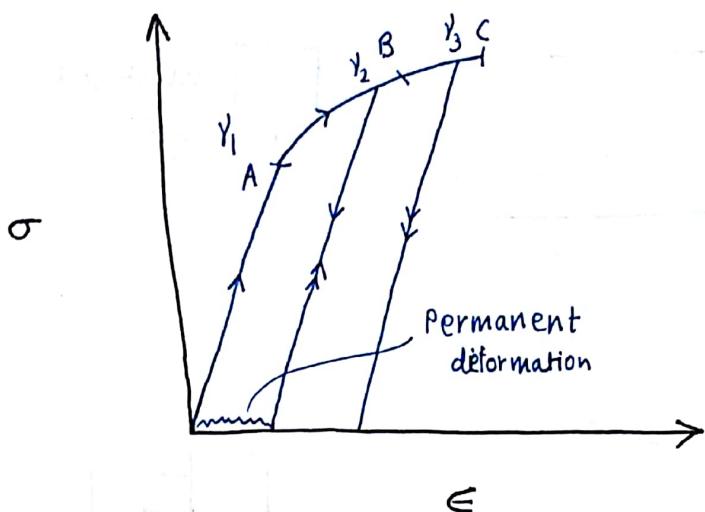
$$\sigma = \frac{P}{A}$$

$$\epsilon_1 = \ln\left(\frac{l_1}{l_0}\right)$$

$$\epsilon_2 = \ln\left(\frac{l_2}{l_1}\right)$$

$$\epsilon_{12} = \ln\left(\frac{l_1}{l_0}\right) + \ln\left(\frac{l_2}{l_1}\right) = \ln\left(\frac{l_2}{l_0}\right)$$

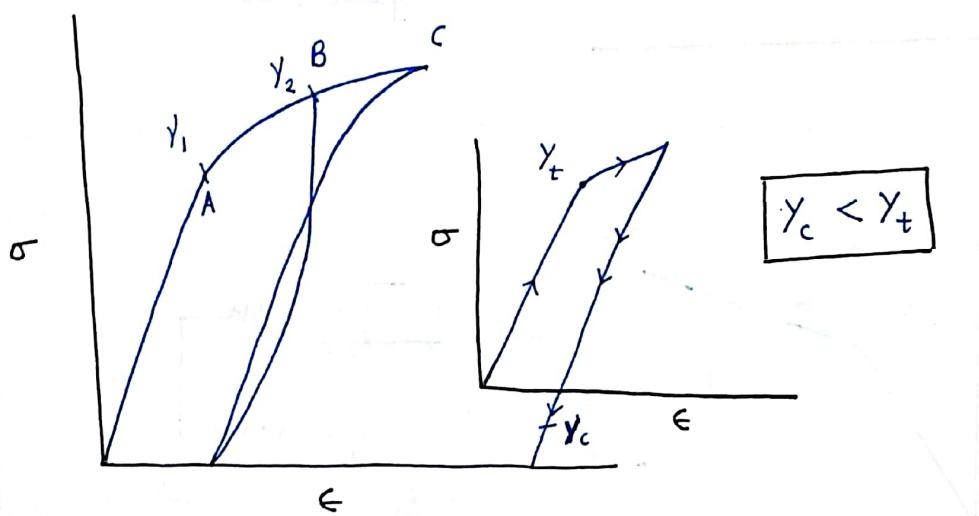
True Stress - Strain



AB = uniform plastic deformation

BC = non-uniform plastic deformation

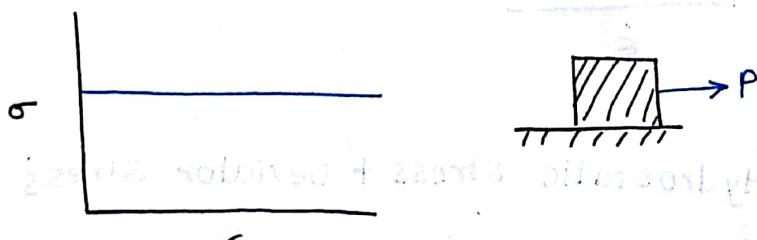
OA = Linear elastic



Bauschinger Effect

Total deformation = elastic + plastic

Materials \Rightarrow Rigid + Perfectly Plastic
elastic + (-)



$$\sigma_{ij} = \sigma'_{ij} +$$

$$s_i = \sigma'_{ij} n_j$$

Stress n

shear

Principle

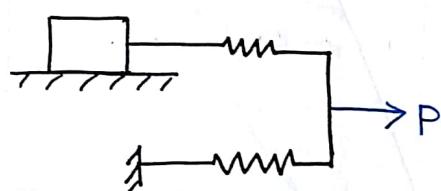
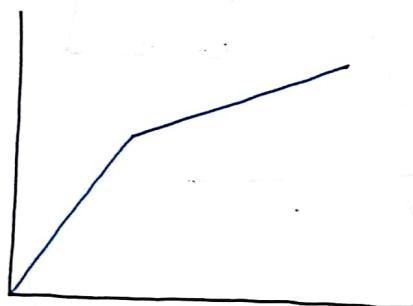
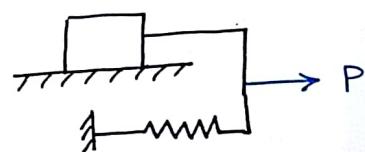
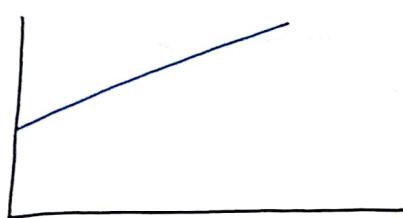
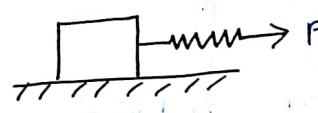
$$\bar{s} =$$

$\Rightarrow S$

$$(\sigma_{11} -$$

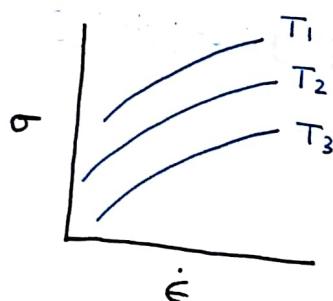
$$\sigma_{12}$$

$$\sigma_{13}$$



$$\epsilon = f(\sigma, T, \dot{\epsilon})$$

$$\dot{\epsilon} \uparrow \Rightarrow \sigma \uparrow$$



$$T_3 > T_2 > T_1$$

$$\text{Total Stress} = \text{Hydrostatic stress} + \text{Deviator stress}$$

$$\left\{ \frac{1}{3} (\sigma_{11} + \sigma_{22} + \sigma_{33}) \right\}$$

change of elastic volume

plastic shear

$$\sigma_{ij} = \sigma'_{ij} + \frac{1}{3} \sigma_{kk} \delta_{ij}$$

$$S_i = \sigma_{ij} n_j$$

Stress normal to the inclined plane, $\sigma_n = \bar{\sigma} \cdot \bar{n}$

$$\Rightarrow \sigma_n = \sigma_{ij} n_j n_i$$

Shear stress components, $\sigma_s = \sqrt{s^2 - \sigma_n^2}$

Principle Stresses

$$\bar{\sigma} = \sigma \bar{n}$$

↳ scalar multiple

$$\Rightarrow S_1 = \sigma n_1, \quad S_2 = \sigma n_2 \quad \& \quad S_3 = \sigma n_3$$

$$\left. \begin{array}{l} (\sigma_{11} - \sigma) n_1 + \sigma_{21} n_2 + \sigma_{31} n_3 = 0 \\ \sigma_{12} n_1 + (\sigma_{22} - \sigma) n_2 + \sigma_{32} n_3 = 0 \\ \sigma_{13} n_1 + \sigma_{23} n_2 + (\sigma_{33} - \sigma) n_3 = 0 \end{array} \right\} \begin{vmatrix} \sigma_{11} - \sigma & \sigma_{21} & \sigma_{31} \\ \sigma_{12} & \sigma_{22} - \sigma & \sigma_{32} \\ \sigma_{13} & \sigma_{23} & \sigma_{33} - \sigma \end{vmatrix} = 0$$

$$\sigma^3 - I_1 \sigma^2 + I_2 \sigma - I_3 = 0$$

$$I_1 = \sigma_{11} + \sigma_{22} + \sigma_{33}$$

$$I_2 = \sigma_{11} \sigma_{22} + \sigma_{22} \sigma_{33} + \sigma_{33} \sigma_{11} - \sigma_{12}^2 - \sigma_{23}^2 - \sigma_{31}^2$$

$$I_3 = \sigma_{11} \sigma_{22} \sigma_{33} + 2 \sigma_{12} \sigma_{23} \sigma_{31} - \sigma_{11} \sigma_{23}^2 - \sigma_{22} \sigma_{31}^2 - \sigma_{33} \sigma_{12}^2$$

(From Deviatoric)



$$\left. \begin{array}{l} J_1 = \\ J_2 = \\ J_3 = \end{array} \right\}$$

$\sigma > \gamma \Rightarrow$ Yielding starts

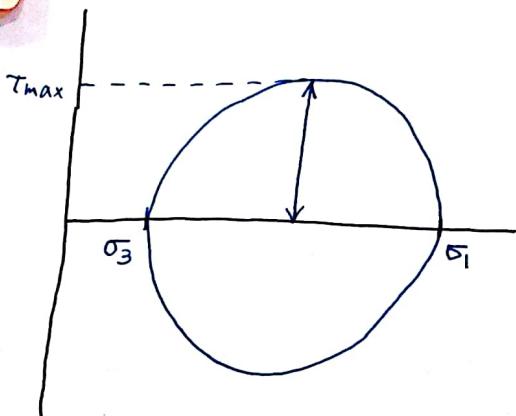
Yield Criteria : TRESCA
von-Mises

Maximum shear stress reaches a critical volume



yielding starts $[\tau_{\max.} = K]$

↳ shear yield stress



$$\tau_{\max.} = \frac{\sigma_1 - \sigma_3}{2}$$

$$\boxed{\sigma_1 - \sigma_3 = 2K}$$

$$\gamma \approx K$$

1) uniaxial loading $\begin{bmatrix} \sigma_1 = \gamma \\ \sigma_2 = 0 \\ \sigma_3 = 0 \end{bmatrix}$

2) pure torsion $\begin{bmatrix} \sigma_1 = K \\ \sigma_2 = 0 \\ \sigma_3 = -K \end{bmatrix}$

Shear strain energy per unit volume → Reaches a critical value



Yielding

$$(\sigma_1 - \sigma_2)^2 + (\sigma_2 - \sigma_3)^2 + (\sigma_3 - \sigma_1)^2 = \text{constant}$$

①

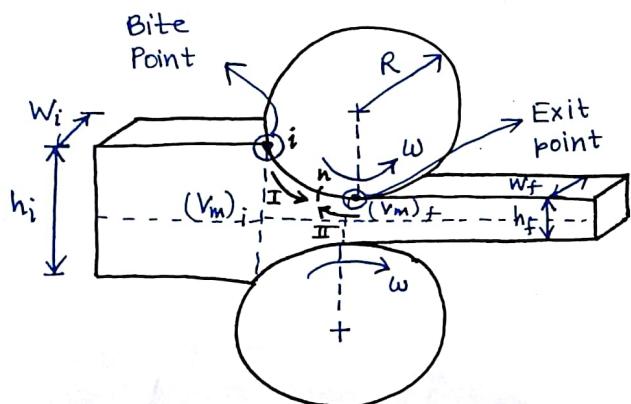
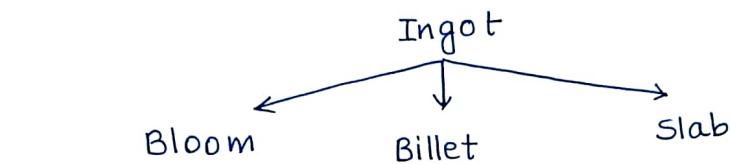
ROLLING

Slab / Plate / Sheet / Strip

Rail / Channel

* $t > 6 \text{ mm} \rightarrow \text{Plate}$

$t < 6 \text{ mm}$ { $w > 600 \text{ mm} \rightarrow \text{sheet}$
 $w < 600 \text{ mm} \rightarrow \text{strip}$



I \rightarrow Lagging Zone

II \rightarrow Outstripping Zone

$$v_R = \omega \times R$$

$$\omega \gg h$$

$$w_i = w_f$$

Plane-strain deformation

$$v_R > (v_m)_i$$

$$(v_m)_f > v_R$$

$$(v_m)_n = v_R$$

yield criteria

$$\sigma_1 - \sigma_3 = 2k$$

$$\sigma_x - (-p) = 2k$$

$$\sigma_x + p = 2k$$

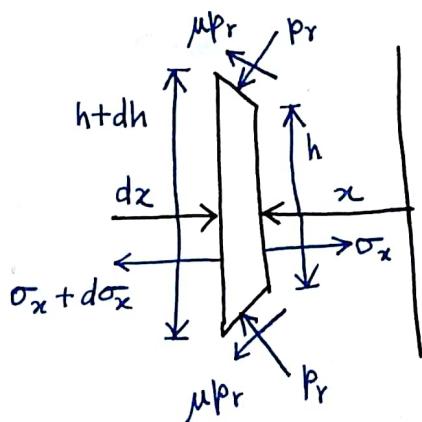
$$\sum f_x = 0$$

$$\sum f_y = 0$$

$$\sigma_1 =$$

$$\sigma_2 =$$

$$-p_r = \sigma_y = -p$$



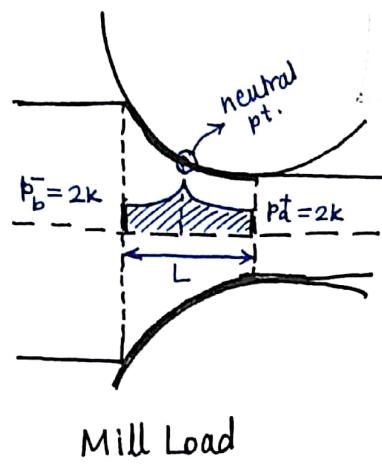
Boundary conditions:

#1 No pull at front & back ends.

$$\sigma_{xa} = 0 \Rightarrow H_a = 0 \text{ at the exit point}$$

$$\text{Exit Point} \Rightarrow p_a^+ = 2k$$

$$\text{Bite Point} \Rightarrow p_b^- = 2k$$



$$\begin{aligned} \text{Front Pull} \Rightarrow F_a \\ \sigma_{xa} \Rightarrow \sigma_x + p = 2k \\ p = 2k - \sigma_{xa} \end{aligned}$$

$$(p_a^+)_{\text{no-front pull}} < (p_a^+)_{\text{with pull}}$$

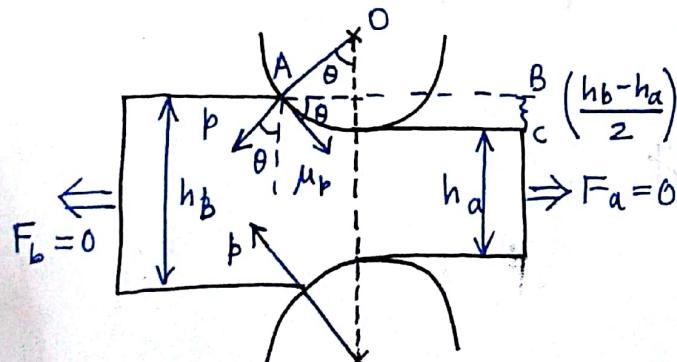
$$\begin{aligned} \sigma_{xb} + p^- = 2k \\ (p_b^-)_{\text{no-back pull}} < (p_b^-)_{\text{with pull}} \end{aligned}$$

Rolling load

$$F = \int_{\alpha_n}^{\theta} p_r() d\theta + \int_0^{\alpha_n} p_r() d\theta$$

$$\text{Torque} = \int \mu p_r() d\theta + \int \mu p_r() d\theta$$

$$\text{Power} = T \cdot \omega$$



Draft for un-aided rolling

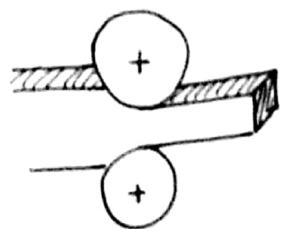
$$h_b - h_a \equiv \Delta h \text{ for unaided rolling}$$

$$p \sin \theta < \mu p \cos \theta$$

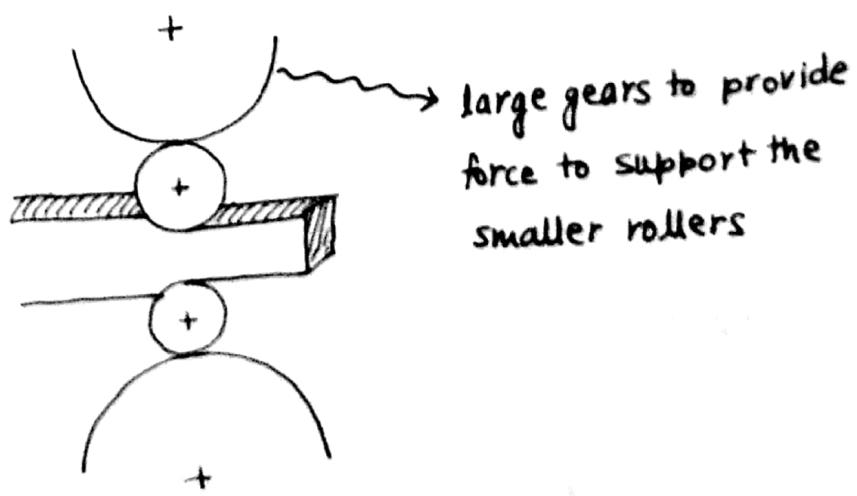
$$\mu \geq \tan \theta$$

1. Single-stand Rolling

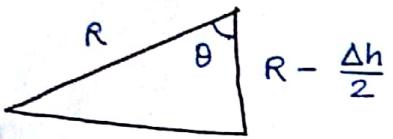
i. Two-high rolling stand



ii. Four-high rolling stand



2. Multi-stage/stand Rolling



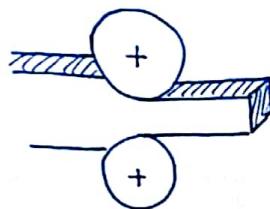
$$\begin{aligned} \tan \theta &= \frac{\left[R^2 - \left(R - \frac{\Delta h}{2} \right)^2 \right]^{1/2}}{R - \frac{\Delta h}{2}} \\ &= \left[R^2 - R^2 - \frac{\Delta h^2}{4} + R \Delta h \right]^{1/2} / \left(R - \frac{\Delta h}{2} \right) \\ &= \frac{R \left\{ \frac{\Delta h}{R} \right\}^{1/2}}{R \left\{ 1 - \frac{\Delta h}{2R} \right\}^{1/2}} = \sqrt{\frac{\Delta h}{R}} = \mu \end{aligned}$$

$\Rightarrow \boxed{\Delta h = \mu^2 R}$

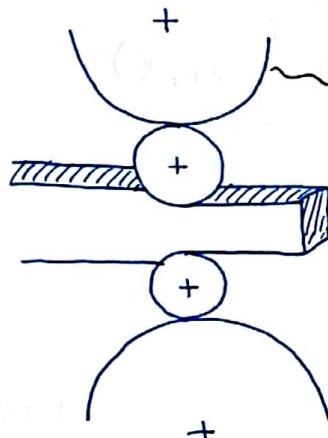
Classification of Rolling Process

1. Single-stand Rolling

i. Two-high rolling stand



ii. Four-high rolling stand

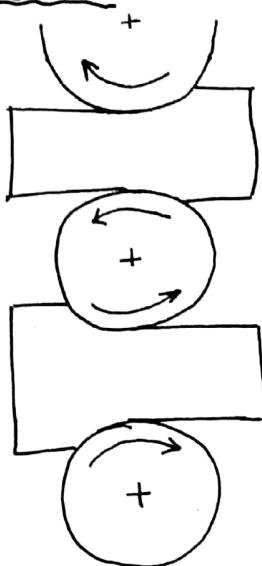


large gears to provide force to support the smaller rollers

2. Multi-stage/ Stand Rolling

3. Cluster-rolling mill

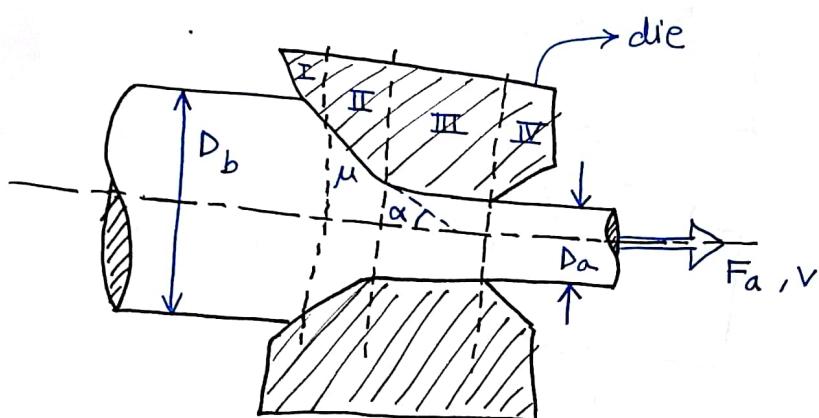
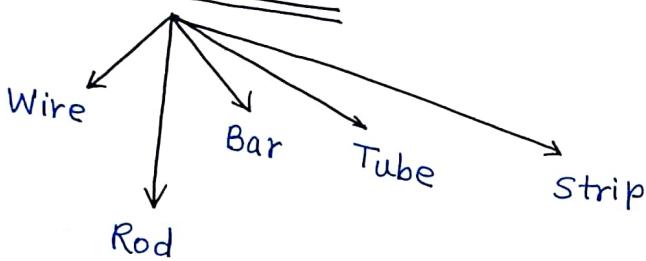
4. Three-high rolling mill



Defects:

- ① Thicker product (but uniform)
- ② Non-uniform thickness
- ③ Waviness / Lack of flatness
- ④ Alligatoring

DRAWING



$$\left(\frac{D_b - D_a}{D_b} \right)$$

* If $\left(\frac{D_b - D_a}{D_b} \right)$ is very high, then heat gen. ΔH is also very high which consequently decreases the life span of the die.

* Steps

1. Elemental stress distribution & its equilibrium cond'n.
2. Yield criteria
3. Boundary cond'n,
4. Get the value of F_a