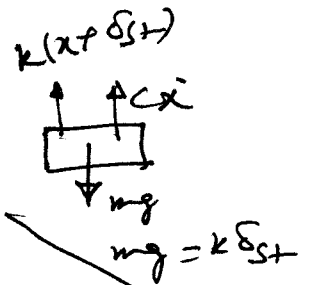
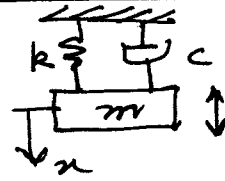


TU/HW-2 (Solutions)

①

Problem ① The system is:

$$k = 1000 \text{ N/m}, m = 5.5 \text{ kg}$$



If $F_d =$ damping force, then, $F_d = c\dot{x}$

When $F_d = 40 \text{ N}$, $\dot{x} = 1 \text{ m/s}$

$$\text{So, } \underline{c = F_d / \dot{x} = 40 \text{ Ns/m}}$$

② The DEOM is: $m\ddot{x} + c\dot{x} + kx = 0$ (~~0 = 0~~)

→ The first step should be computation of ζ . $\zeta = \frac{c}{2\sqrt{km}} = 0.2697 < 1$

Hence, the system is underdamped.

$$\text{So, } x(t) = X_0 e^{(-\zeta\omega_n t)} \sin(\omega_d t + \phi)$$

$$\text{or } x(t) = e^{(-\zeta\omega_n t)} [A \sin \omega_d t + B \cos \omega_d t]$$

Note → You may use either form.
→ Be patient with computations, they are ~~are~~ often a bit lengthy.

$$\text{Here } \omega_n = \sqrt{\frac{k}{m}} = 13.484 \text{ rad/s}, \zeta\omega_n = 3.637$$

$$\omega_d = \omega_n \sqrt{1 - \zeta^2} = \underline{\underline{12.984 \text{ rad/s}}}$$

$$\text{Then } x = e^{(-3.637t)} (A \sin 12.984t + B \cos 12.984t)$$

To evaluate A & B, we use initial conditions --- (1)

$$x(0) = 50 \times 10^{-3} \text{ m}, \quad \dot{x}(0) = 0$$

$$\text{Then, (1) gives: } \underline{\underline{50 \times 10^{-3} = B}} \text{ (in m)}$$

$$\begin{aligned} \text{Now, } \dot{x} &= e^{(-3.637t)} (A \times 12.984 \cos 12.984t - 12.984 B \sin 12.984t) \\ &\quad - 3.637 e^{(-3.637t)} (A \sin 12.984t + B \cos 12.984t) \end{aligned}$$

--- (2)

(2)

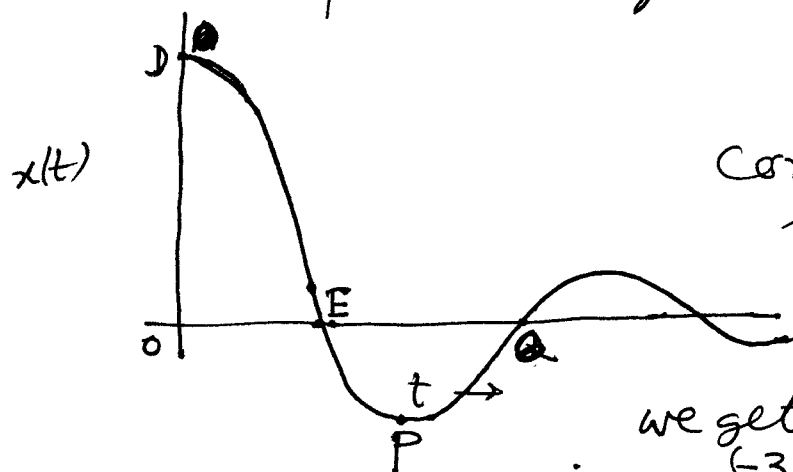
Then $\dot{x}(0) = 0$ gives: $0 = 12.984A - 3.637B$
 So, $A = 14.0 \times 10^{-3} \text{ m}$

Thus, the required expression for displacement is:

$$x(t) = e^{(-3.637t)} [14 \sin 12.984t + 50 \cos 12.984t] \quad \text{--- (II)}$$

check mm.

Part (b) [A sketch of $x(t)$ versus t may help visualize the situation.]



Note that point P corresponds to the highest point.

Substituting values of A & B in (I),

we get:

$$\dot{x} = e^{(-3.637t)} [-0.70012 \sin 12.984t] \quad \text{m/s}$$

At P, $\dot{x} = 0 \Rightarrow \sin 12.984t = 0.$

\Rightarrow

So $12.984t = 0, \pi, 2\pi$ etc.

Obviously, P corresponds to $t = \frac{\pi}{12.984} \text{ s} = 0.24195 \text{ s}$
 ($t=0$ corresponds to point A) $\approx 0.242 \text{ s}$

Putting this value of t in (II), we get

$$x = -20.741 \times 10^{-3} \text{ m} = -20.741 \text{ mm}.$$

Hence, required total distance moved
 $= 50 + 20.741 = 70.741 \text{ mm}$ & time taken
 is $0.242 \text{ s}.$

Part c Here time corresponding to point E is required. At E, $x=0$. From (II), we get
 $\tan 12.984t = -3.5714$ ~~$\Rightarrow 12.984t = 105.64^\circ$~~
 $\Rightarrow 12.984t = 105.64 \times \frac{\pi}{180} \Rightarrow t = 0.142 \text{ s}.$

Problem 2 (a) $\delta = \ln\left(\frac{14.4}{1.2}\right) \approx \ln\left(\frac{1.2}{2.1}\right)$

Hence, $\delta = 2.485$

(b) $\delta = \frac{2\pi\eta}{\sqrt{1-\eta^2}} \Rightarrow \eta = \frac{\delta}{\sqrt{4\pi^2 + \delta^2}} = 0.368$

(c) $k_t = \frac{GJ}{L}$, $G = 34.5 \times 10^9 \text{ N/m}^2$, $L = 0.4 \text{ m}$,
 $J = \frac{\pi d^4}{32} = \frac{\pi \times (9 \times 10^{-3})^4}{32} \text{ m}^4$

$\therefore k_t = 55.55 \text{ N-m/rad}$, $I_d = 0.6 \text{ kgm}^2$

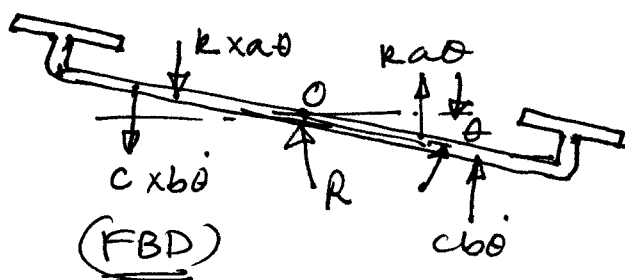
So, $\omega_n = \sqrt{\frac{k_t}{I_d}} = 9.622 \text{ rad/s}$

(d) $\omega_d = \omega_n \sqrt{1-\eta^2} = 8.95 \text{ rad/s}$

(e) $\Gamma_{\text{damping}} = c_t \dot{\theta}$ Hence, damping torque at unit angular velocity is nothing but c_t only.

Now, $\eta = \frac{c_t}{2I_d \omega_n} \Rightarrow c_t = 4.25 \text{ Nms/rad}$.

Problem 3 This problem is a little tricky, be careful.



Let θ (+ive CW) be the generalized coordinate.

Here $I_0 = 0.01 \text{ kgm}^2$

$a = 0.08 \text{ m}$, $b = 0.1 \text{ m}$,

$k = 5 \times 10^3 \text{ N/m}$, $c = 70 \text{ Ns/m}$

$T = \frac{1}{2} I_0 \dot{\theta}^2$, $V = 2 \times \frac{1}{2} k(a\theta)^2$, $D = 2 \times \frac{1}{2} c(b\dot{\theta})^2$

\Rightarrow The DEOM is: $I_0 \ddot{\theta} + 2c b^2 \dot{\theta} + 2k a^2 \theta = 0$

(4)

$$\therefore \omega_n = \sqrt{\frac{2ka^2}{I_0}} = 80 \text{ rad/s}, \quad \beta = \frac{2cb^2}{2I_0\omega_n} = 0.875$$

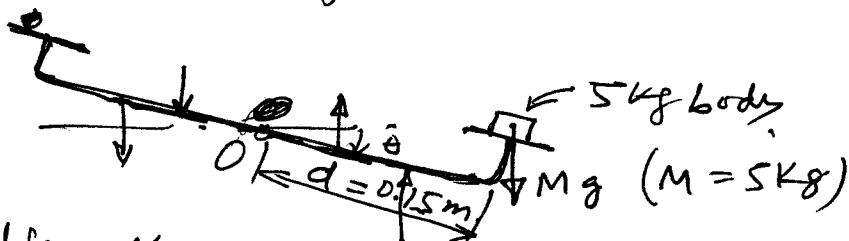
After 5 kg mass is placed:-

We have a new system with 5 kg extra mass on righthand pan. We treat it as a particle & hence, now $I_0' =$ new moment of inertia of this new system about an axis through

$$O = I_0' = 0.01 + 5 \times (0.15)^2 = 0.1225 \text{ kgm}^2.$$

→ A new equilibrium position results at $\theta = \theta_0$. Show that (by taking moments about O in the FBD for this new system) $\theta_0 = 0.115 \text{ rad}$.

→ We may take a new generalized coordinate measured from this θ_0 line. But we can also continue to measure rotation from horizontal position only. You will see that this will introduce an extra constant term in the DEOM & make it nonhomogeneous, as follows:-



$$\text{New KE} = T' = \frac{1}{2} I_0' \dot{\theta}^2, \quad D' = D = 2 \times \frac{1}{2} c (b\dot{\theta})^2,$$

$$V' = V - Mg \times d \times \theta \text{ (Note)} = 2 \times \frac{1}{2} k (a\theta)^2 - Mg d \theta$$

Then, the DEOM will be: $I_0' \ddot{\theta} + 2cb^2 \dot{\theta} + 2ka^2 \theta = Mg d \quad (1)$

Then, $\zeta' = \zeta_{\text{new}} = \frac{2Cb^2}{2\sqrt{2ka^2 \times I_0'}} = 0.25 < 1.$

$$\omega_n' = \sqrt{\frac{2ka^2}{I_0'}} = 22.86 \text{ rad/s.}$$

$$\zeta \omega_n' = 5.714 \text{ \& } \omega_d' = \omega_n' \sqrt{1 - \zeta'^2} = 22.13 \text{ rad/s}$$

Hence, $\theta(t) = e^{(-5.714t)} \left[A \sin 22.13t + B \cos 22.13t \right] + \frac{Mgd}{K}$

→ Find $\dot{\theta}(t)$

→ Use initial conditions: $\theta(0) = 0$
& $\dot{\theta}(0) = 0.$

Do the Computations in detail. These should give $A = -0.02969$
& $B = -0.11496.$

Then, ~~θ~~ $\dot{\theta} = 0$ would give

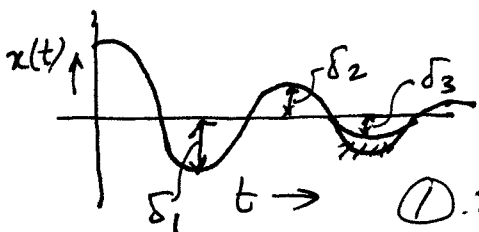
$$22.13t = 0, \pi, 2\pi \dots \text{etc.}$$

We have to take $22.13t = \pi$

$$\Rightarrow t = 0.14196 \text{ s.}$$

Then, $\theta(0.14196) = 0.166 \text{ rad} = \underline{\underline{9.51^\circ}}$

Problem 4 $I_d = 50 \times 10^{-3} \times (12 \times 10^{-3})^2 = 72 \times 10^{-7} \text{ kg m}^2$
 $\sigma_d = \frac{2\pi}{\omega_d} = 4.2 \text{ s} \Rightarrow \omega_d = 1.496 \text{ rad/s}$



$$\frac{\delta_1}{\delta_2} = \frac{90}{12}, \frac{\delta_2}{\delta_3} = \frac{90}{12} \Rightarrow \frac{\delta_1}{\delta_3} = \left(\frac{90}{12}\right)^2$$

(Not to scale) ① ∴ Logarithmic decrement $= \delta = \ln\left(\frac{\delta_1}{\delta_3}\right)$
 $\Rightarrow \delta = 2 \ln\left(\frac{90}{12}\right) = \frac{2\pi\zeta}{\sqrt{1-\zeta^2}} \Rightarrow \underline{\underline{\zeta = 0.5399}}$

② $\omega_n = \frac{\omega_d}{\sqrt{1-\zeta^2}} = 1.7774 \text{ rad/s}$

③ $\omega_n = \sqrt{\frac{K_t}{I_d}} \Rightarrow K_t = \omega_n^2 I_d = 22.75 \times 10^{-6} \text{ Nm/rad} \Rightarrow$

$$\textcircled{4} \quad C_t = 2\mathcal{I}_d \omega_n \left[\because \mathcal{I} = \frac{C_t}{2\mathcal{I}_d \omega_n} \right]$$

$$= 13.82 \times 10^{-6} \text{ Nm/rad/s}$$

Part (b)

$$\theta = e^{-\gamma \omega_n t} (A \sin \omega_d t + B \cos \omega_d t)$$

$$(i) \quad = e^{(-0.95956t)} [A \sin 1.496t + B \cos 1.496t]$$

$$\dot{\theta} = e^{(-0.95956t)} [1.496A \cos 1.496t - 1.496B \sin 1.496t]$$

$$(ii) \quad -0.95956 e^{(-0.95956t)} [A \sin 1.496t + B \cos 1.496t]$$

(Patience, patience please!)

→ Let $\dot{\theta}(0) = \omega_0$, the unknown initial angular velocity generated by the angular impulse. (Such a short duration impulse generates a finite angular velocity, but no appreciable angular displacement, do you visualize?)

So, $\theta(0) = 0$.

Also, if T is the time reqd. to swing through 25° , then

$$\dot{\theta}(T) = 0 \text{ when } \theta(T) = \frac{25 \times \pi}{180} \text{ rad.}$$

Using $\theta(0) = 0$ & $\dot{\theta}(0) = \omega_0$ in (i) & (ii),

you should get ~~also~~ $B = 0$, $A = \frac{\omega_0}{1.496}$

Also, $\dot{\theta}(T) = 0$ & $\theta(T) = \frac{25\pi}{180} \text{ rad}$

would yield $T = 0.6684 \text{ s}$, $A = 0.98478$

& $\omega_0 = 1.4732 \text{ rad/s}$

Finally, the energy supplied by the impulse

(7)

goes into making ^{the initial} kinetic energy of the rotor
 of magnitude $\frac{1}{2} I_d \omega_0^2 = 7.813 \times 10^{-6} \text{ J}$.
 Hence, the energy supplied by the
 impulse = $7.813 \mu\text{J}$

Problem (5) The bag strikes the platform
 with velocity = $\sqrt{2gh} = \sqrt{2 \times 9.81 \times 0.4} \text{ m/s}$

Conservation of momentum gives

$$870 V = 70 \sqrt{2 \times 9.81 \times 0.4}$$

$$\Rightarrow V = 0.2254 \text{ m/s, which is our } \dot{x}(0).$$

Assuming inelastic collision, the
 bag + platform is the new system
 mass & there will be a new
equilibrium position, note. However,
 we can measure x from earlier
 equilibrium position only. This
 gives the following DEOM:

$$870 \ddot{x} = -160 \times 10^3 x - 500 \dot{x} + 70 \times 9.81$$

$$\Rightarrow \ddot{x} + 0.5747 \dot{x} + 114.94 x = 0.7893$$

--- (a)

Now note the following interesting point:-

Let the DEOM be $m\ddot{x} + c\dot{x} + kx = F$

Then, $\ddot{x} + \frac{c}{m}\dot{x} + \frac{k}{m}x = F/m$

But $\frac{c}{m} = 2\gamma\omega_n$, $\frac{k}{m} = \omega_n^2$. Hence,

$\ddot{x} + 2\gamma\omega_n \dot{x} + \omega_n^2 x = F/m$ — (b) Now
 compare (a) & (b). Then, $2\gamma\omega_n = 0.5747$ &

$$\omega_h^2 = 114.94 \Rightarrow \omega_h = 10.721 \text{ rad/s}$$

$$2\gamma\omega_h = 0.5747 \Rightarrow \gamma = 0.0268 < 1.$$

→ It is not essential to do it this way, but the form (b) is used in many text books etc. & you should be familiar with it!

Then, $\omega_d = 10.717 \text{ rad/s}$

Next, note that the P.I. of (c) is

$$\text{simply } x_p = \frac{0.7893}{114.94} = 6.867 \times 10^{-3}$$

Then, the general solution of (a) is:-

$$x = 6.867 \times 10^{-3} + e^{(-0.28735t)} [A \sin 10.717t + B \cos 10.717t]$$

Now, $x(0) = 0$, $\dot{x}(0) = 0.2254 \text{ m/s}$.

These would give $A = 0.02085$,

$$B = -0.006867$$

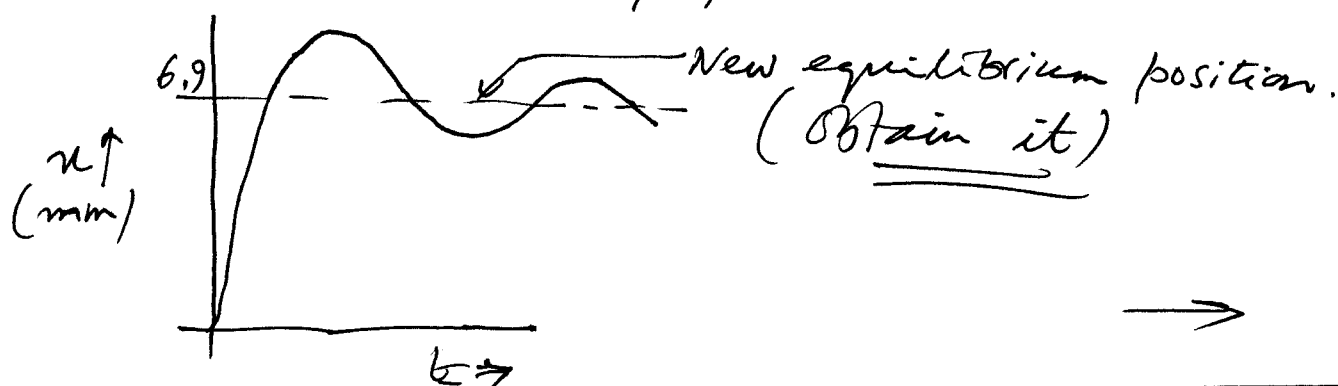
→ At the lowest position, $\dot{x} = 0$

$$\Rightarrow \tan 10.717t = -3.3343$$

$$\Rightarrow t = 0.174 \text{ s.}$$

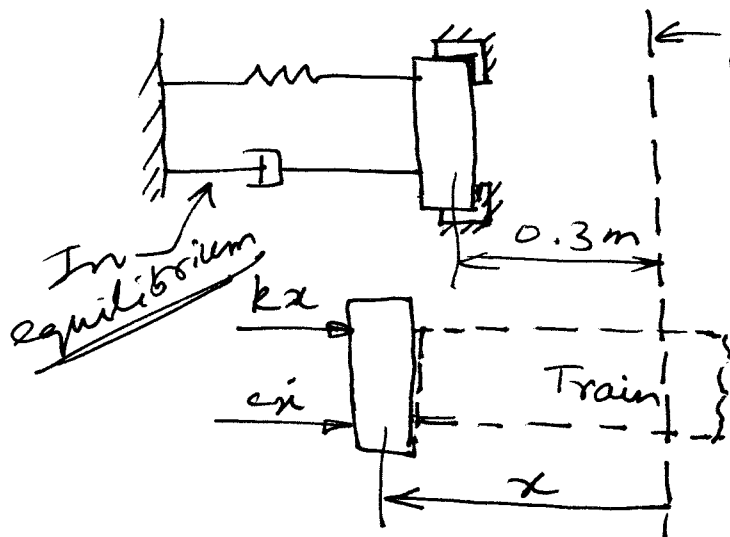
Carry out the details

Then, $x(0.174 \text{ s}) = 27.7 \text{ mm.}$



Problem 6

Note that the spring initial compression is provided by some pre-compression device like the one shown in the figure below:



← position of buffer with no pre-compression.

If x is measured the way shown, the DEOM becomes

$$\ddot{x} + 2.4\dot{x} + 0.6x = 0,$$

check this!

(You may use a different x , of course, measured from static eqbm position of buffer)

Then, $\omega_n = \sqrt{0.6} = 0.7746 \text{ rad/s},$

$$2\zeta\omega_n = 2.4 \Rightarrow \zeta = 1.55 > 1$$

→ So, this system is overdamped.

$$\text{So, } x(t) = A e^{(-\zeta + \sqrt{\zeta^2 - 1})\omega_n t} + B e^{(-\zeta - \sqrt{\zeta^2 - 1})\omega_n t}$$

etc.

Now Complete the solution

END of Tu-2, Part I