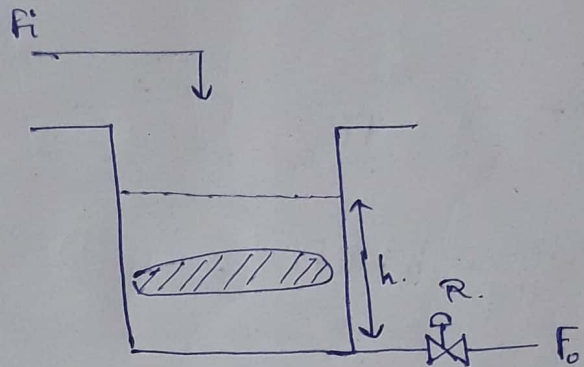


Mathematical Modeling

- i) Theoretical method.
- ii) Empirical method
- iii) Semiempirical method.

Liquid storage tanks



F_i = Inlet volumetric flowrate (m^3/sec)

F_o = Outlet volumetric flowrate.

h = liquid level.

A = Cross-sectional area.

R = Resistance to the outlet flow.

According to -

the mass conservation law,

Rate of accumulation of liquid mass = inlet flowrate - outlet flowrate

$$\frac{d(\rho V)}{dt} = \rho_i F_i - \rho_o F_o$$

$$\rho = \rho_i = \rho_o$$

$$\therefore \frac{dV}{dt} = F_i - F_o$$

Tumi Robe Niroke /

$$\frac{d(Ah)}{dt} = F_i - F_o$$

$$F_o = \frac{h}{R}$$

$$\frac{Adh}{dt} = F_i - \frac{h}{R}$$

$$\Rightarrow \frac{Adh}{dt} + \frac{h}{R} = F_i$$

$$\Rightarrow AR \frac{dh}{dt} + h = RF_i \quad \text{--- (1)}$$

At steady state condition,

$$h = h_s, F_i = F_{is}$$

$$0 + h_s = RF_{is} \quad \text{--- (2)}$$

$$AR \frac{d(h - h_s)}{dt} + (h - h_s) = R(F_i - F_{is}) \quad [1 - 2]$$

$$\text{Let, } h' = h - h_s \text{ \& } F_i - F_{is} = F_i'$$

$$\therefore AR \frac{dh'}{dt} + h' = RF_i'$$

$$\therefore AR sH'(s) + H'(s) = RF_i'(s)$$

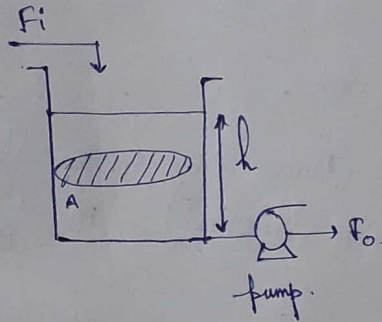
$$\therefore \frac{H'(s)}{F_i'(s)} = \frac{R}{(ARs + 1)}$$

Here,

$$\text{Gain of the process } (K_p) = R.$$

$$\tau_p = AR.$$

Liquid storage tank with constant outlet:-



$$\frac{d}{dt} (Ah) = F_i - F_o.$$

$$\Rightarrow \frac{A dh}{dt} = F_i - F_o.$$

• At steady state condition,

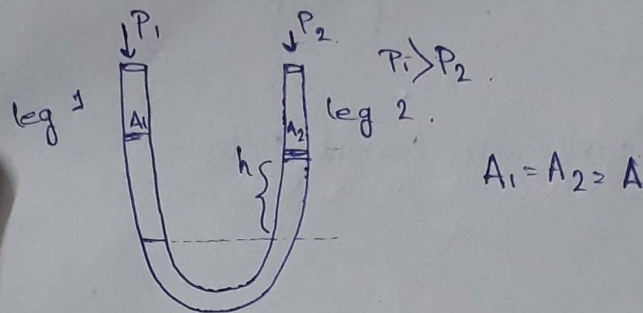
$$0 = F_{is} - F_{os} \quad \text{Here, } F_{os} = F_o.$$
$$= F_{is} - F_o.$$

$$\frac{A dh'}{dt} = F_i'$$

$$As H'(s) = F_i'(s).$$

$$\Rightarrow \frac{H'(s)}{F_i'(s)} = 1/As.$$

U-tube Manometer



Force balance equation,

$$A_1 P_1 - P_2 A_2 - h \rho g - (\text{force due to the fluid friction}) = m \frac{d^2 h}{dt^2}$$

$$A \Delta P - h \rho g - \frac{8 \mu L}{R^2} A \frac{dh}{dt} = m \frac{d^2 h}{dt^2}$$

$$m \frac{d^2 h}{dt^2} + \frac{8 \mu L}{R^2} A \frac{dh}{dt} + A h \rho g = A \Delta P$$

$$\Rightarrow \frac{m}{A \rho g} \frac{d^2 h}{dt^2} + \frac{8 \mu L}{R^2 \rho g} \frac{dh}{dt} + h = \frac{\Delta P}{\rho g}$$

$$\Rightarrow \frac{\rho A L}{A \rho g} \frac{d^2 h}{dt^2} + \frac{8 \mu L}{R^2 \rho g} \frac{dh}{dt} + h = \frac{\Delta P}{\rho g} \quad \left[\because m = \rho A L \right]$$

$$\Rightarrow \frac{L}{g} \frac{d^2 h}{dt^2} + \frac{8 \mu L}{R^2 \rho g} \frac{dh}{dt} + h = \frac{\Delta P}{\rho g}$$

$$\Rightarrow \tau^2 \frac{d^2 h}{dt^2} + 2 \zeta \tau \frac{dh}{dt} + h = \frac{\Delta P}{\rho g}$$

$$\tau^2 s^2 + 2\zeta \tau s + 1 = 0$$

$$\tau^2 s^2 H(s) + 2\zeta \tau s H(s) + H(s) = \frac{\Delta P}{\rho g}$$

is

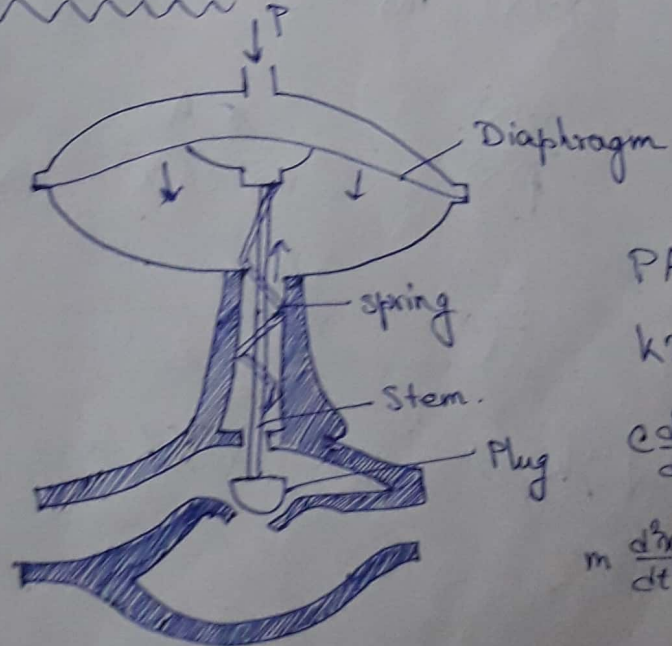
$$\tau = \sqrt{\frac{4\mu L}{g}} \quad ; \quad 2\zeta \tau = \frac{8\mu L}{R^2 \rho g}$$

$$\zeta = \frac{4\mu L}{R^2 \rho g \tau}$$

$$= \frac{4\mu L}{R^2 \rho g} \cdot \sqrt{\frac{g}{4\mu L}}$$

$$\frac{H(s)}{\Delta P(s)} = \frac{1/\rho g}{\tau^2 s^2 + 2\zeta \tau s + 1}$$

Pneumatic Control Valve



PA ↓

kx ↑

$c \frac{dx}{dt} \uparrow$

$m \frac{d^2x}{dt^2}$

Force-balance equation,

$$PA - kx - c \frac{dx}{dt} = m \frac{d^2x}{dt^2}$$

$$\Rightarrow \frac{PA}{k} - x - \frac{c}{k} \frac{dx}{dt} = \frac{m}{k} \frac{d^2x}{dt^2}$$

$$\Rightarrow \frac{m}{k} \frac{d^2x}{dt^2} + x + \frac{c}{k} \frac{dx}{dt} = \frac{PA}{k}$$

$$\Rightarrow \frac{m}{k} s^2 X(s) + X(s) + \frac{cs}{k} X(s) = \frac{\frac{PA}{k} \frac{Ps(s)}{k}}{AP(s)}$$

$$\Rightarrow ms^2 X(s) + kX(s) + csX(s) = \frac{PA(s)}{AP(s)}$$

$$\Rightarrow X(s) \{ ms^2 + k + cs \} = \frac{Ps(A) PA(s)}{AP(s)}$$

$$\Rightarrow \frac{X(s)}{P(s)} = \frac{A}{ms^2 + cs + k}$$

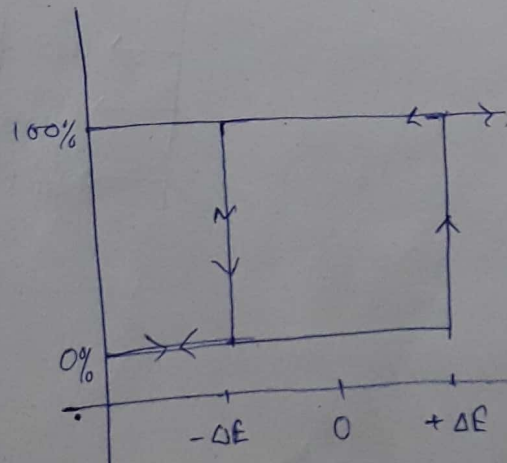
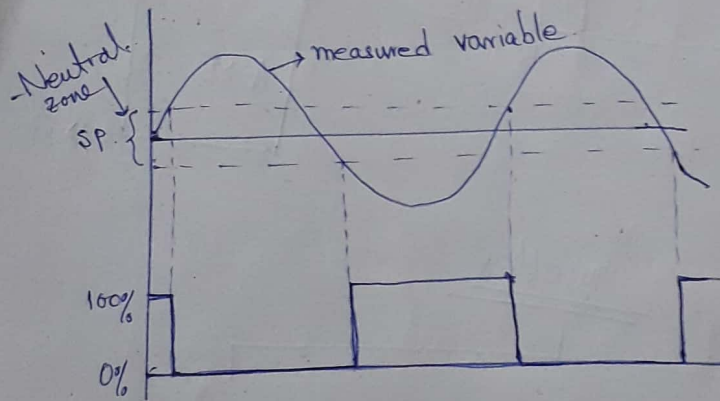
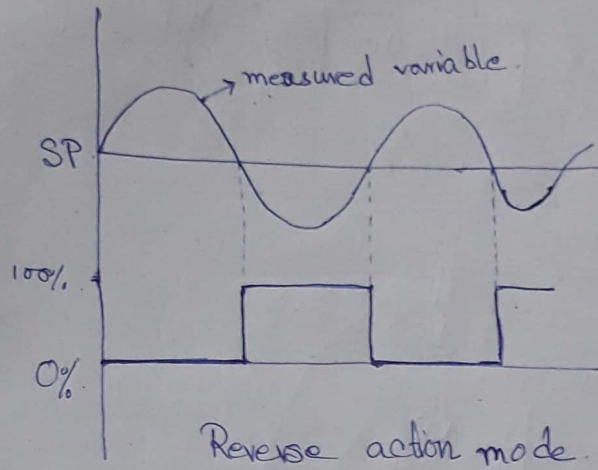
$$\Rightarrow \frac{X(s)}{P(s)} = \frac{A/k}{\frac{m}{k}s^2 + \frac{c}{k}s + 1}$$

$$\zeta = \sqrt{m/k}$$

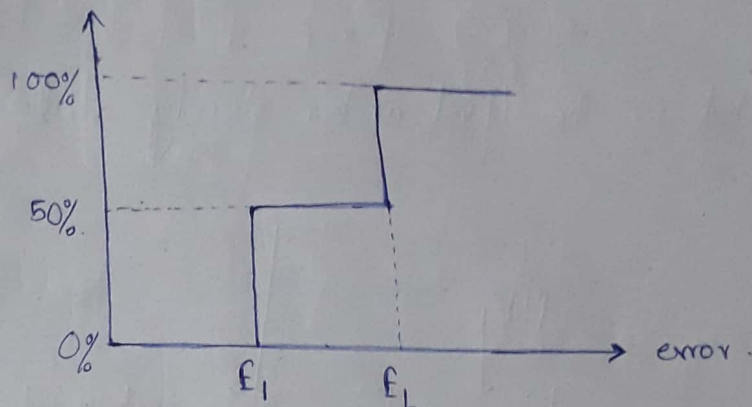
$$m \ll k$$

ON/OFF Controller

JAI SHREE RAM!



Three - position Controller



Proportional Controller

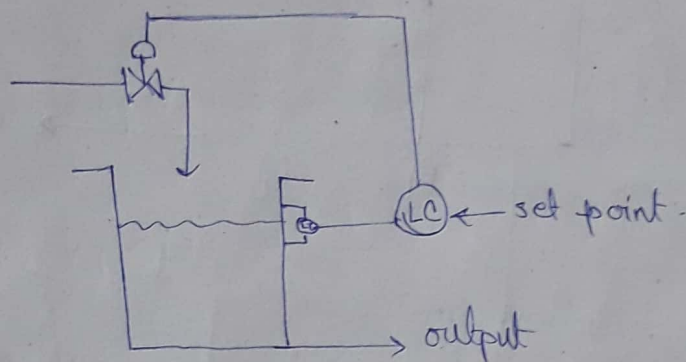
$$m(t) = K_p e(t)$$

↙ gain

$$\text{Proportional band} = \frac{100\%}{K_p}$$

$$PB = \frac{\text{percentage change in error}}{\text{percentage change in o/p}}$$

What is the level under steady state control?



given

Range of LT : 0 to 70 inch

Level Controller : P only

PB = 75%

Bias = 50%

Set point = 40 inches

Load : fixed at 3.5 gpm

Valve : Pr. drop ΔP is constant

Linear characteristics delivers 6 gpm
at 100% ~~stroke~~ stroke.

Soln

$$\text{Inlet flow} = \frac{3.5}{6} \times 100\%$$

$$m = 58.33 \%$$

$$\text{Set point} = \frac{40}{70} \times 100\%$$

$$= \cancel{57.14} 57.14\%$$

$$= x \cdot [\text{koto saat cha thakbo}]$$

$$m = kp e(t) + 50\%$$

$$= \frac{100}{PB} (x - C) + 50\%$$

$$58.33 = \frac{100}{75} (57.14 - C) + 50$$

$$\Rightarrow \cancel{57.14 - C} \quad C = 50.8925 \%$$

$$C \approx 51\%$$

$$\text{output Controlled variable in inches} = \frac{51}{100} \times 70$$

$$= 35.7 \text{ inches}$$

6 gpm
stroke