

Stability Robustness for Multi-Input Systems

(Read sections 5.4, 5.5 and 5.6)

In Lab 3 we work with the 2 DOF AERO plant, which is a 2-input system. There are two motor-driven propellers that influence the yaw and pitch motions.

How do we compute "stability margins" for a multi-input system? One thought would be to insert individual perturbations (eg. g_1, g_2, \dots) on each of the plant inputs, and let each of the g_i s vary in gain or phase one at a time. The drawback to this approach is that a control system can have low sensitivity to perturbations on individual plant inputs but be very sensitive to small perturbations that occur simultaneously on multiple inputs.

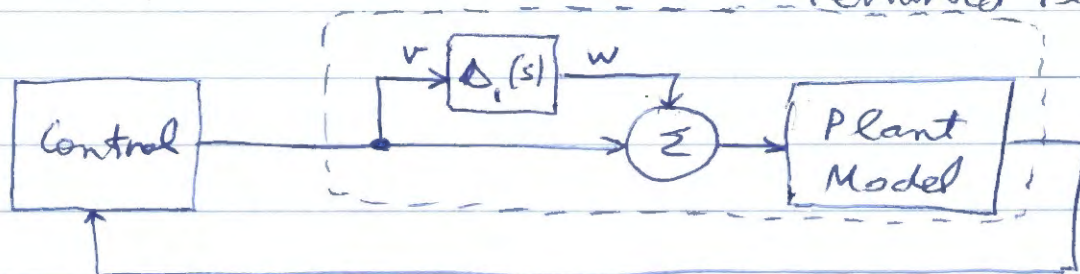
Modern Stability Robustness

The modern approach works for both multi-input systems as well as single-input systems (which are a special case).

This approach considers two different ways of perturbing a plant model, as shown on the next page.

First Perturbation Model

Perturbed Plant Model



The plant is a p -input system.

$\Delta_i(s)$ represents a stable, p -input, p -output unknown system. If $\Delta_i(s) = 0$ the closed-loop system is stable by design. The closed-loop system remains stable provided that

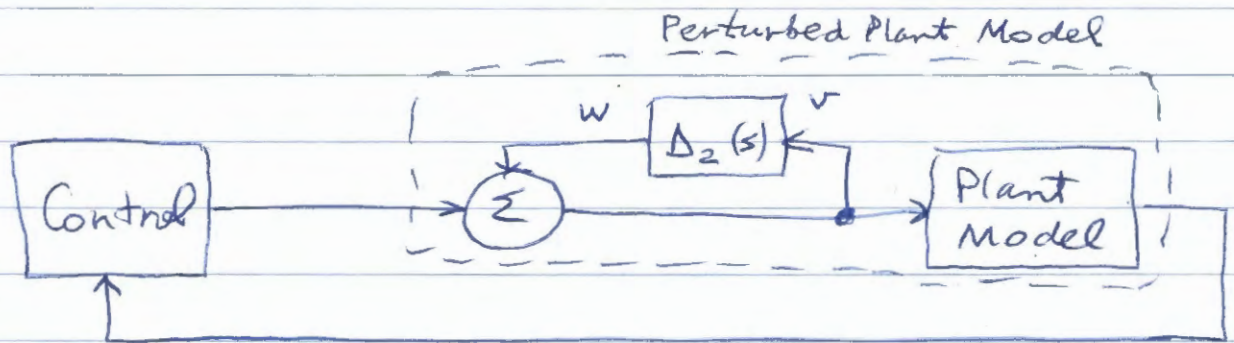
$$\|\Delta_i(s)\|_{\infty} < \delta_i.$$

called the "system infinity norm", is a measure of the "size" of the perturbation. Includes simultaneous gain and phase perturbations on all plant inputs.

δ_i is a number, called a "stability robustness bound" that can be computed for any control system

We want δ_i to be as large as possible. A specific guideline will be given on the next page, after introducing the second perturbation model.

Second Perturbation Model



This closed-loop system will remain stable if $\|\Delta_2(s)\|_\infty < \delta_2$.

δ_2 is the second stability robustness bound

For control systems in which the plant and/or controller has a pole at $s=0$, or for any unstable plant, δ_1 and δ_2 cannot be larger than 1.

most useful control systems.

Guideline: We would like $\min(\delta_1, \delta_2) \geq 0.5$

Modern stability robustness bounds provide bounds on classical stability margins (ie the stability margins cannot be too small):

$$\hat{g}_{\max} = \max\left(1 + \delta_1, \frac{1}{1 - \delta_2}\right)$$

$$\hat{g}_{\min} = \min\left(1 - \delta_1, \frac{1}{1 + \delta_2}\right)$$

If the plant has gain perturbations g_1, \dots, g_p on the plant inputs, the control system remains stable if $\hat{g}_{\min} < g_i < \hat{g}_{\max}, i=1, \dots, p$

Also, $\hat{\phi}_{\max} = 2 \sin^{-1} \left(\frac{\max(\delta_1, \delta_2)}{2} \right)$

If the plant has phase perturbations $e^{-j\phi_1}, \dots, e^{-j\phi_p}$ on the plant inputs, the control system remains stable if

$$|\phi_i| < \hat{\phi}_{\max}, \quad i=1, \dots, p$$

Note that, for a single-input control system,

classical $\xrightarrow{\text{upper gain margin}} \hat{g}_{\max} > \hat{g}_{\max}$

classical lower $\{ \hat{g}_{\min} < \hat{g}_{\min}$
gain margin

classical $\{ \hat{\phi}_{\max} > \hat{\phi}_{\max}$
phase margin

A Note on Classical Gain Margins

Recall that upper and lower gain margins are reported in dB:

$$UGM = 20 \log_{10}(\hat{g}_{\max})$$

$$LGM = 20 \log_{10}(\hat{g}_{\min})$$

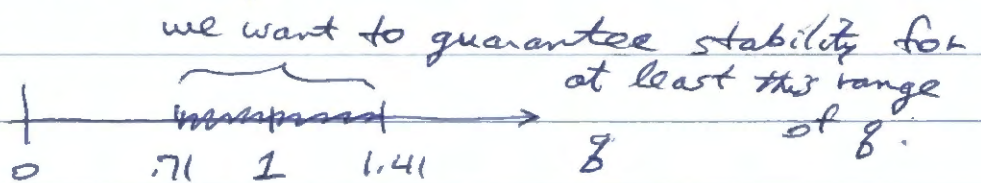
These expressions can be inverted by dividing both sides by 20, then raising both sides as exponents of 10:

$$g_{\max} = 10^{\frac{UGM}{20}} \quad \left(\begin{array}{l} \text{recall} \\ 10^{\log_{10}(x)} = x \end{array} \right)$$

$$g_{\min} = 10^{\frac{LGM}{20}}$$

$$\text{So } UGM = 3 \text{ dB} \Rightarrow g_{\max} = 10^{3/20} = 1.41$$

$$LGM = -3 \text{ dB} \Rightarrow g_{\min} = 10^{-3/20} = .71$$



What kind of classical gain margin estimates do we get from a modern stability robustness analysis when δ_1 and δ_2 both equal the smallest desired value of 0.5? From page (47)

$$\hat{g}_{\max} = \max \left(1 + 0.5, \frac{1}{1 - 0.5} \right) = 2$$

$$\hat{g}_{\min} = \min \left(1 - 0.5, \frac{1}{1 + 0.5} \right) = 0.5$$

These estimates are better than ± 3 dB and they are valid for all plant inputs simultaneously.

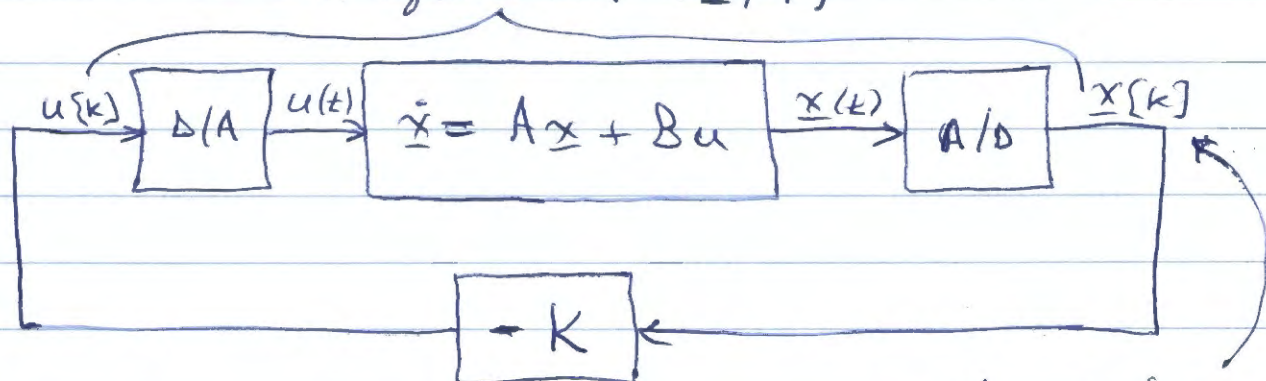
How about the phase margin estimate?

$$\hat{\phi}_{\max} = 2 \sin^{-1} \left(\frac{.5}{2} \right) = 29^\circ$$

Very close to the classical guideline and it is valid for all plant inputs simultaneously.

Observers (Chapter 7)

Motivation: if we want to control a plant by state feedback, we need to measure all of the state variables. For example, a digital state-feedback regulator:
 ZOH equivalent (Φ, Γ)



have to measure all the state variables

Recall, for the cart-pendulum system the state variables are:

- x_1 - pendulum angular position - measured w/ encoder
- x_2 - " " velocity - not measured!
- x_3 - motor angular position - measured w/ encoder
- x_4 - " " velocity - not measured!