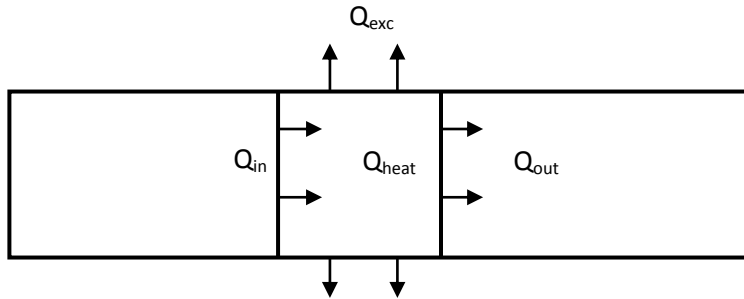


## Transient analysis of temperature distribution in a disk

1. Consider the heat balance at thin circular layer in a disk with thickness of  $\Delta r$  from time  $t$  to  $t+\Delta t$



$$Q_{in} = Q_{out} + Q_{heat} + Q_{exc}$$

$Q_{in} = -Sk \partial u(t, r)/\partial t \Delta t$  – heat, that comes from left side

$Q_{out} = -Sk \partial u(t, r+\Delta r)/\partial t \Delta t$  - heat, that goes to right side

$Q_{heat} = CS\Delta r\Delta u$  – heat that goes for heating of layer

$Q_{exc} = 4\pi r\Delta r W(u-\Theta)$  – heat, that goes to air through exchange

where:

$u(t, r)$  – temperature distribution function as function of time and radius

$S = 2\pi rh$  – cross sectional area of layer,  $h$  – disc thickness

$k$  – material heat conductivity

$C$  – material heat capacity

$\Delta u$  – temperature change in a layer

$W$  – heat transfer coefficient with ambient air

$\Theta$  – air temperature

After conversion we get such **heat equation**

$$a u_{rr} = u_t + b (u - \Theta) \quad (1)$$

where

$a = k/C$  - parameter, that characterize thermal conductivity

$u_{rr} = (1/r \partial u/\partial r + \partial^2 u/\partial r^2)$  – second derivative of  $u$  in cylindric coordinates

$u_t = \partial u/\partial t$  – first time derivative of  $u$

$b = 2W/Ch$  - parameter, that characterize heat exchange to air

We have such **boundary and initial conditions**:

at  $r = r_{int}$   $T=400 \text{ K} = \text{const}$ ,

at  $r = r_{ext}$  heat transfer should be neglected, mean that  $\partial u/\partial r|_{r_{ext}} = 0$

at  $t = 0$   $u(r, 0) = \Theta$ , disc was at equilibrium with air

2. To solve this equation we use a method of finite differences. Radius and time should take discrete values with step  $\Delta r$  and  $\Delta t$  respectively, and represent as a vectors  $r(j)$  and  $t(n)$ . Temperature  $u(t, r)$  will represent as a matrix  $u(n, j)$ . Let's write derivatives in term of finite differences:

$$\partial u / \partial r = [ u(n, j+1) - u(n, j) ] / \Delta r$$

$$\partial^2 u / \partial r^2 = [ u(n, j+1) - 2 u(n, j) + u(n, j-1) ] / \Delta r^2$$

$$\partial u / \partial t = [ u(n+1, j) - u(n, j) ] / \Delta t$$

After replacing equation (1) should rewrite in such form

$$u(n+1, j) = A u(n, j) + B u(n, j+1) + C u(n, j-1) + D \Theta \quad (2)$$

where

$$A = 1 - a\Delta t / r\Delta r - 2a a\Delta t / \Delta r^2 - b\Delta t$$

$$B = a\Delta t / r\Delta r + a\Delta t / \Delta r^2$$

$$C = a\Delta t / \Delta r^2$$

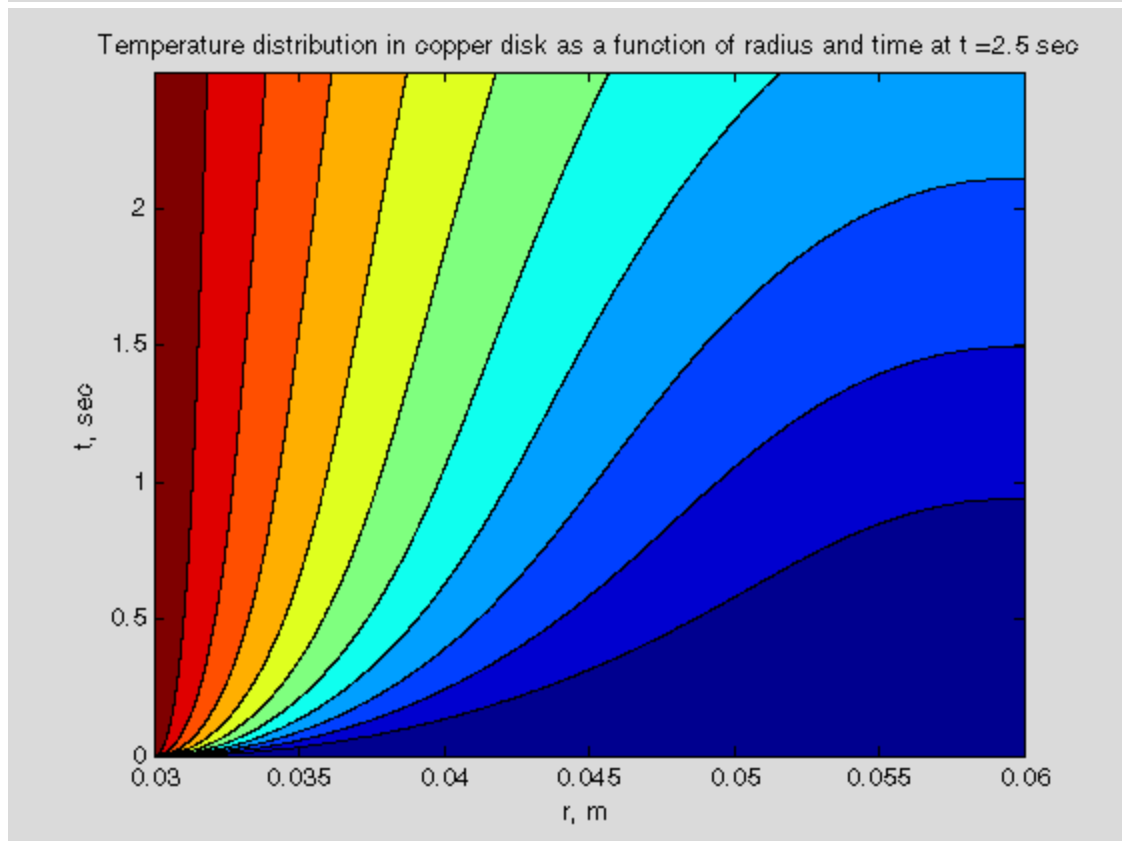
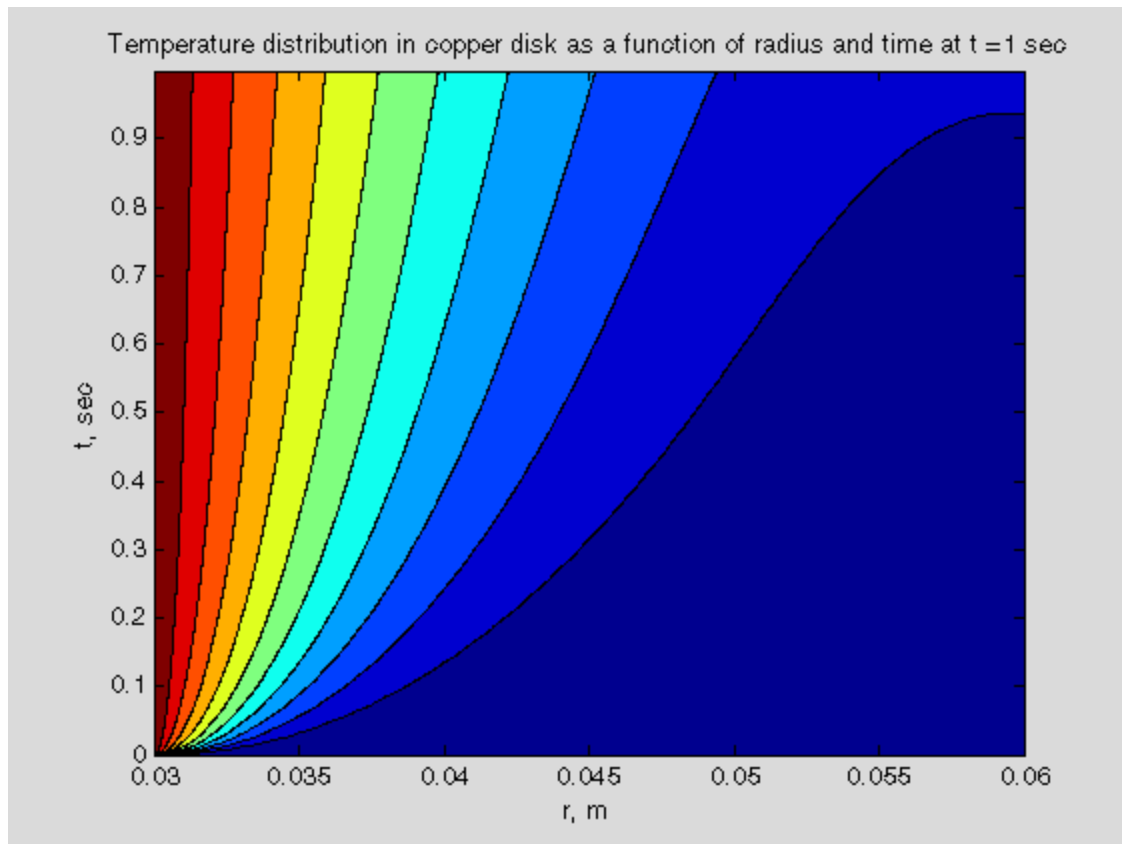
$$D = b\Delta t$$

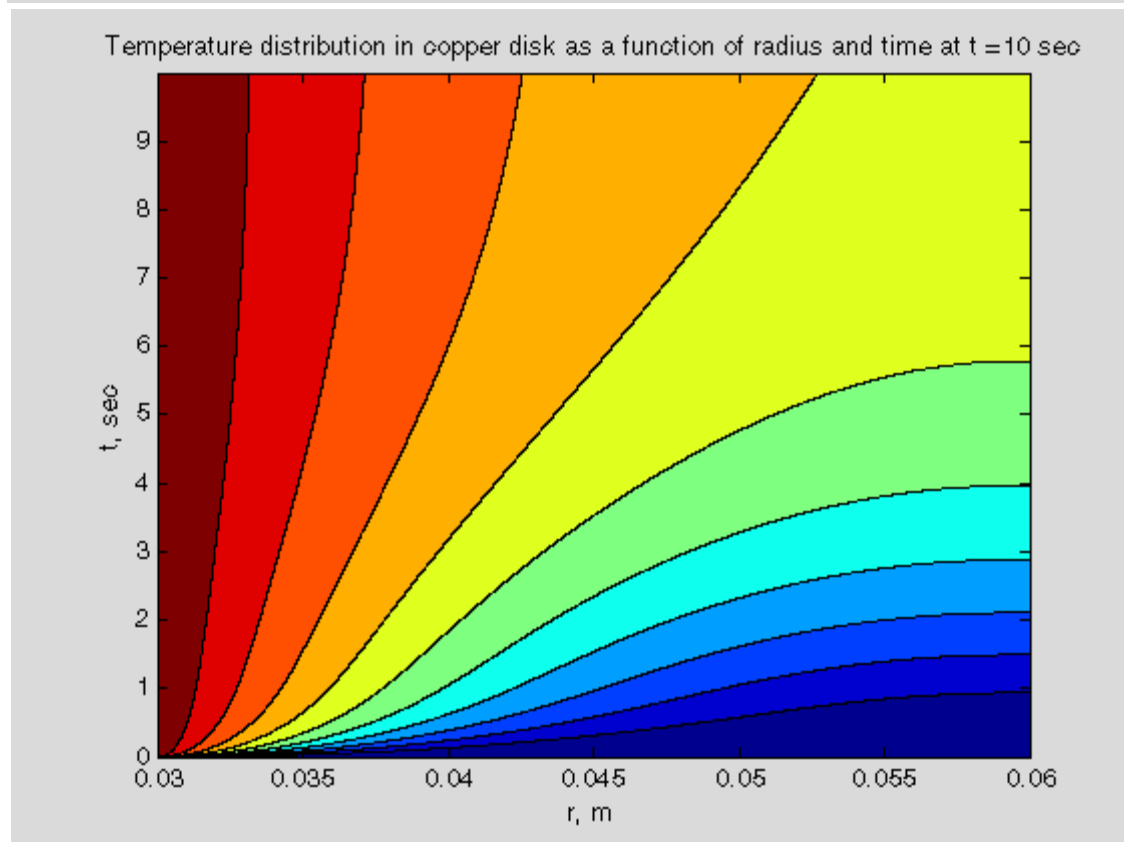
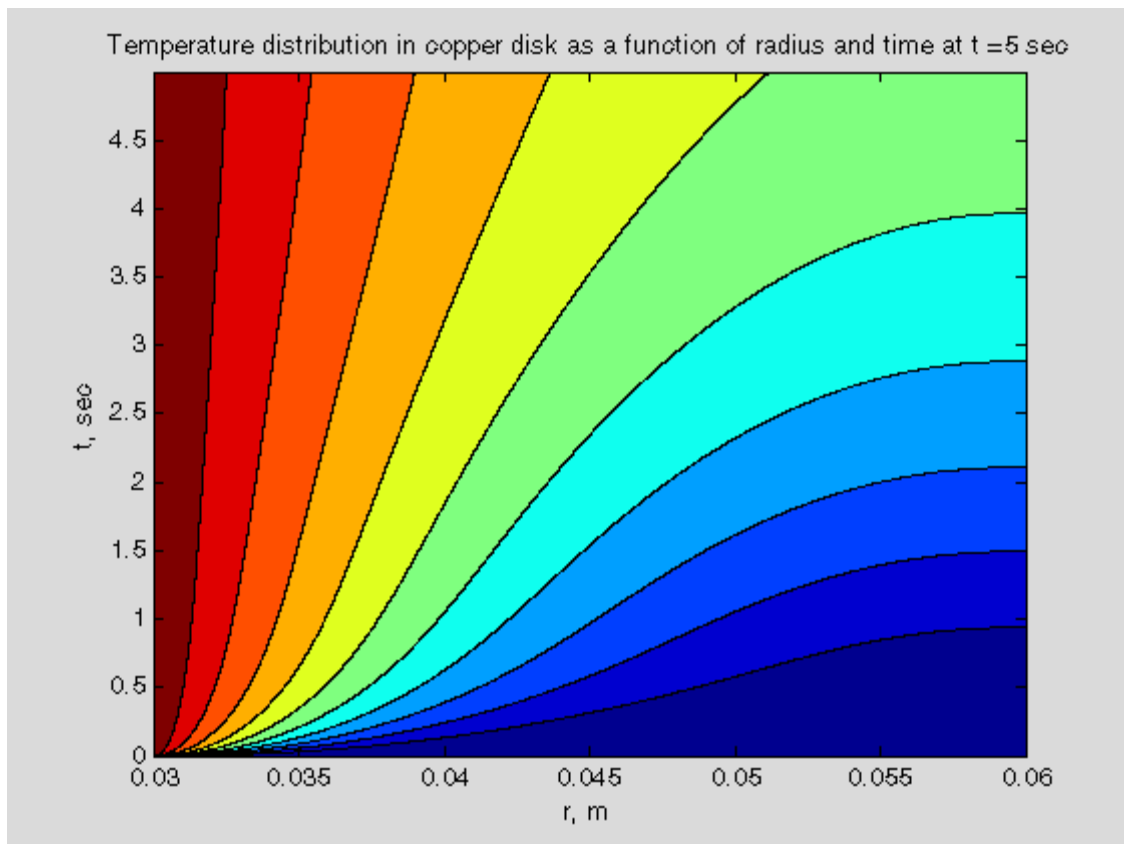
3. For modeling we need to choose value of  $\Delta r$  and  $\Delta t$ . For  $\Delta r = 10^{-3} m$  coefficients should equal to such values:

$\Delta t, sec$	$10^{-3}$	$10^{-4}$	$10^{-5}$
A	0.76	0.98	$\sim 1$
B	0.12	0.012	$1.2 \cdot 10^{-3}$
C	0.12	0.012	$1.2 \cdot 10^{-3}$
D	$\sim 10^{-4}$	$\sim 10^{-5}$	$\sim 10^{-6}$

4. Scripts and functions, developed in MATLAB:
- initParameters* – initiate parameters of system
  - heatDisk* (*a*, *b*, *t*, *r*, *init*, *int\_temp*, *air\_temp*, *transfer*) – returns temperature distribution as a matrix *u*
  - showDistribution*( *u\_vals* , *r\_vals* , *t\_vals* , *t\_moment* ) - returns temperature distribution at time moment *t\_moment*
  - animateCylindricDistribution*( *u\_vals* , *r\_vals* , *t\_vals* , *temp\_limits*, *steps* , *contour* ) – returns contour or 3D animation of temperature distribution
  - animateDistribution*( *u\_vals* , *r\_vals* , *t\_vals* , *steps* , *new\_figure* ) – returns contour 2D animation of temperature distribution
  - evalDistribution* – evaluate all developed functions and script *initParameters*

5. Here are presented obtained distributions at different moments of time. Color changed every 10 K.





6. For obtain all graphics and animations evaluate script 'evalDistribution'.