

Calculations in Transport Phenomena

THEORETICAL BACKGROUND

Forms of derivatives :

(1) Partial time derivative, $\frac{\partial c}{\partial t}$.

This means variation of c (the variable) with time with respect to a fixed position (x, y, z) in space.

(2) Total time derivative, $\frac{dc}{dt}$

$$\frac{dc}{dt} = \frac{\partial c}{\partial t} + \frac{\partial c}{\partial x} \cdot \frac{dx}{dt} + \frac{\partial c}{\partial y} \cdot \frac{dy}{dt} + \frac{\partial c}{\partial z} \cdot \frac{dz}{dt}$$

where, $\frac{dx}{dt}$, $\frac{dy}{dt}$ and $\frac{dz}{dt}$ are the variation of x, y and z respectively with respect to time.

(3) Substantial time derivative, $\frac{Dc}{Dt}$

It is a special kind of total time derivative also, called 'derivative following the motion'. The expression is

$$\frac{Dc}{Dt} = \frac{\partial c}{\partial t} + v_x \frac{\partial c}{\partial x} + v_y \frac{\partial c}{\partial y} + v_z \frac{\partial c}{\partial z}$$

where, v_x, v_y and v_z are components of local fluid velocity, v .

Equation of continuity :

This is based on conservation of mass. According to this,

Rate of change of density = Divergence of mass flux

Mathematically,

$$\frac{\partial \rho}{\partial t} = -(\nabla \cdot \rho v) \quad \dots(1)$$

or
$$\frac{D\rho}{Dt} = -\rho (\nabla \cdot v)$$

For steady state,
$$\frac{D\rho}{Dt} = 0 \quad \dots(2)$$

θ -component :

$$\rho \left(\frac{\partial v_\theta}{\partial t} + v_r \frac{\partial v_\theta}{\partial r} + \frac{v_\theta}{r} \cdot \frac{\partial v_\theta}{\partial \theta} + \frac{v_\varphi}{r \sin \theta} \cdot \frac{\partial v_\theta}{\partial \varphi} + \frac{v_r v_\theta}{r} - \frac{v_\varphi^2 \cot \theta}{r} \right)$$

$$= -\frac{1}{r} \cdot \frac{\partial p}{\partial \theta} + \mu \left(\nabla^2 v_\theta + \frac{2}{r^2} \cdot \frac{\partial v_r}{\partial \theta} - \frac{v_\theta}{r^2 \sin^2 \theta} - \frac{2 \cos \theta}{r^2 \sin^2 \theta} \cdot \frac{\partial v_\varphi}{\partial \varphi} \right) + \rho g_\theta \quad \dots(15)$$

φ -component :

$$\rho \left(\frac{\partial v_\varphi}{\partial t} + v_r \cdot \frac{\partial v_\varphi}{\partial r} + \frac{v_\theta}{r} \cdot \frac{\partial v_\varphi}{\partial \theta} + \frac{v_\varphi}{r \sin \theta} \cdot \frac{\partial v_\varphi}{\partial \varphi} + \frac{v_\varphi v_r}{r} + \frac{v_\theta v_\varphi}{r} \cot \theta \right)$$

$$= -\frac{1}{r \sin \theta} \cdot \frac{\partial p}{\partial \varphi} + \mu \left(\nabla^2 v_\varphi - \frac{v_\varphi}{r^2 \sin^2 \theta} + \frac{2}{r^2 \sin^2 \theta} \frac{\partial v_r}{\partial \varphi} \right. \\ \left. + \frac{2 \cos \theta}{r^2 \sin^2 \theta} \cdot \frac{\partial v_\theta}{\partial \varphi} \right) + \rho g_\varphi \quad \dots(16)$$

In equations (14) to (16),

$$\nabla^2 = \frac{1}{r^2} \frac{\partial}{\partial r} \left(r^2 \frac{\partial}{\partial r} \right) + \frac{1}{r^2 \sin \theta} \cdot \frac{\partial}{\partial \theta} \left(\sin \theta \frac{\partial}{\partial \theta} \right) + \frac{1}{r^2 \sin^2 \theta} \left(\frac{\partial^2}{\partial \varphi^2} \right) \quad \dots(17)$$

Shell Energy Balance :

For steady state, a statement of the law of conservation of energy gives

$$\left(\begin{array}{c} \text{rate of thermal} \\ \text{energy in} \end{array} \right) - \left(\begin{array}{c} \text{rate of thermal} \\ \text{energy out} \end{array} \right) + \left(\begin{array}{c} \text{rate of thermal} \\ \text{energy production} \end{array} \right) = 0 \quad \dots(18)$$

Thermal energy may enter or leave a system

- (i) by virtue of heat conduction
- (ii) by fluid motion (convective transport).

Thermal energy may be produced in various ways viz.

- (1) by degradation of mechanical energy
- (2) by degradation of electrical energy
- (3) by conversion of chemical energy into heat.

Equation (18) can be applied for a system consisting of a thin slab or shell, the thickness of which may be allowed to approach zero.

Example 1. With the help of momentum balance, derive expressions for the following in case of a steady laminar flow of incompressible fluid in a pipe

(a) Velocity profile

(b) Average velocity

(c) Volumetric flow.

the top plate is moving with a constant velocity 'U'. Derive expressions for velocity distribution and shear stress at the wall.

Solution.

From momentum balance,

$$\frac{\partial \tau}{\partial y} = -\frac{\partial p}{\partial x}$$

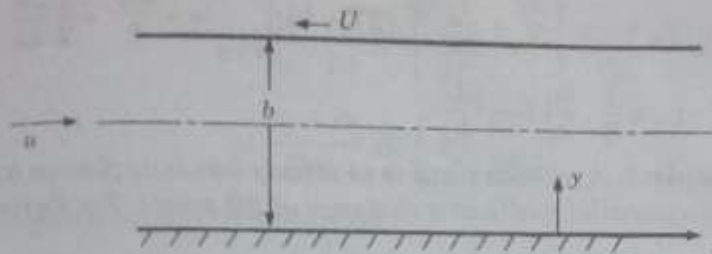


Fig. 9.2

Also, $\tau = -\mu \frac{\partial u}{\partial y}$

So, $\mu \frac{\partial^2 u}{\partial y^2} = \frac{\partial p}{\partial x} \quad \therefore \quad \frac{\partial^2 u}{\partial y^2} = \frac{1}{\mu} \cdot \frac{\partial p}{\partial x}$

Integration w.r.t 'Y' gives, $\frac{\partial u}{\partial y} = \frac{1}{\mu} \frac{\partial p}{\partial x} Y + C_1$

Further integration w.r.t. 'Y', gives,

$$u = \frac{1}{\mu} \cdot \frac{\partial p}{\partial x} \cdot \frac{Y^2}{2} + C_1 Y + C_2$$

Boundary conditions :

(i) When $Y = 0$, $u = 0$

(ii) When $Y = b$, $u = U$

With the first boundary condition, $C_2 = 0$, and the second boundary condition yields

$$C_1 = \frac{U}{b} - \frac{1}{\mu} \frac{\partial p}{\partial x} \cdot \frac{b}{2}$$

Putting values of C_1 and C_2 in the velocity profile expression,

$$\begin{aligned} u &= \frac{1}{\mu} \frac{\partial p}{\partial x} \cdot \frac{Y^2}{2} + \frac{U}{b} Y - \frac{1}{\mu} \frac{\partial p}{\partial x} \frac{Y}{2} Y \\ &= \frac{UY}{b} + \frac{1}{2\mu} \frac{\partial p}{\partial x} (Y^2 - by) \end{aligned}$$

The expression for velocity distribution is

$$u = \frac{UY}{b} + \frac{1}{2\mu} \frac{\partial p}{\partial x} (Y^2 - by)$$

Shear stress at the wall :

$$\tau_w \Big|_{Y=0} = -\mu \left(\frac{\partial u}{\partial y} \right)_{Y=0}$$

$$\frac{\partial u}{\partial y} = \frac{U}{b} + \frac{1}{2\mu} \frac{\partial p}{\partial x} (2Y - b)$$

$$\tau_w = -\mu \left[\frac{U}{b} + \frac{1}{2\mu} \frac{\partial p}{\partial x} (2Y - b) \right]_{Y=0} = -\frac{\mu U}{b} + \frac{1}{2} \frac{\partial p}{\partial x} \cdot b$$

$$\text{So, } \tau_w = \frac{1}{2} \cdot \frac{\partial p}{\partial x} b - \frac{\mu U}{b}$$

Example 3. A viscous fluid is in steady laminar flow in a slit formed by two parallel walls at a distance of $2B$ apart. Find expressions for

- (a) Velocity profile (b) Shear stress distribution
(c) Volumetric flow rate of fluid.

Solution :

From shell momentum balance,

$$\frac{\partial \tau}{\partial x} = -\frac{\partial p}{\partial z}$$

$$\text{Also, } \tau = -\mu \left(\frac{\partial u}{\partial x} \right)$$

$$\text{So, } \mu \frac{\partial^2 u}{\partial x^2} = \frac{\partial p}{\partial z}$$

$$\therefore \frac{\partial^2 u}{\partial x^2} = \frac{1}{\mu} \frac{\partial p}{\partial z}$$

Integration w.r.t. 'x' gives,

$$\frac{\partial u}{\partial x} = \frac{1}{\mu} \frac{\partial p}{\partial z} x + C_1$$

$$\text{When } x = 0, u = u_{max} \therefore \frac{\partial u}{\partial x} = 0.$$

This is satisfied when $C_1 = 0$.

$$\therefore \frac{\partial u}{\partial x} = \frac{1}{\mu} \cdot \frac{\partial p}{\partial z} x$$

Further integration w.r.t. 'x' gives,

$$u = \frac{1}{\mu} \cdot \frac{\partial p}{\partial z} \frac{x^2}{2} + C_2$$

$$\text{When } x = B, u = 0$$

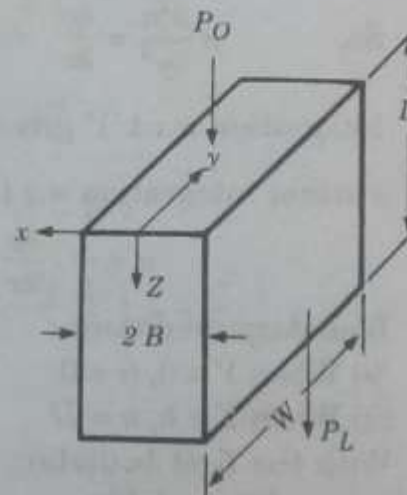


Fig. 9.3

This condition yields, $C_2 = -\frac{1}{\mu} \frac{\partial p}{\partial z} \cdot \frac{B^2}{2}$

Putting value of C_2 in velocity distribution equation,

$$\begin{aligned} u &= \frac{1}{2\mu} \cdot \frac{\partial p}{\partial z} x^2 - \frac{1}{2\mu} \frac{\partial p}{\partial z} B^2 \\ &= \frac{1}{2\mu} \cdot \frac{\partial p}{\partial z} B^2 \left(\frac{x^2}{B^2} - 1 \right) = -\frac{1}{2\mu} \cdot \frac{\partial p}{\partial z} B^2 \left(1 - \frac{x^2}{B^2} \right) \\ &= \frac{P_0 - P_L}{2\mu L} B^2 \left(1 - \frac{x^2}{B^2} \right) \end{aligned}$$

Shear stress distribution :

$$\begin{aligned} \tau &= -\mu \left(\frac{\partial u}{\partial x} \right) \\ \frac{\partial u}{\partial x} &= -\frac{1}{2\mu} \cdot \frac{\partial p}{\partial z} (-2x) = \frac{1}{2\mu} \cdot \frac{\partial p}{\partial z} 2x = \frac{1}{\mu} \cdot \frac{\partial p}{\partial z} x \\ \tau &= -\frac{\partial p}{\partial z} \cdot x = \frac{P_0 - P_L}{L} x \end{aligned}$$

Volumetric flow rate of fluid :

$$\begin{aligned} \text{Average velocity} = u_{av} &= \frac{\int_0^W \int_{-B}^B u \, dx \, dy}{\int_0^W \int_{-B}^B dx \, dy} \\ &= \frac{W \int_{-B}^B \frac{1}{2\mu} \frac{\partial p}{\partial z} (x^2 - B^2) \, dx}{W \int_{-B}^B dx} = \frac{\frac{1}{2\mu} \cdot \frac{\partial p}{\partial z} \left(\frac{x^3}{3} - B^2 x \right) \Big|_{-B}^B}{\left(x \right) \Big|_{-B}^B} \\ &= \frac{P_0 - P_L}{3\mu L} B^2 \end{aligned}$$

Volumetric flow rate = $u_{av} \times \text{Area}$

$$= \frac{P_0 - P_L}{3\mu L} B^2 \times 2BW = \frac{2}{3\mu} \frac{P_0 - P_L}{L} B^3 W$$