

CHAPTER

5

MECHANICS OF MATERIALS

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Lecture Notes:

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Analysis and Design of Beams for Bending

Lecture 11 02/14/2018
Modified from Original

HW Problems Week 6 (due Mon 02/26):

5.11, 5.15, 5.16, 5.53, 5.56, 5.57

Relations Among Load, Shear, and Bending Moment

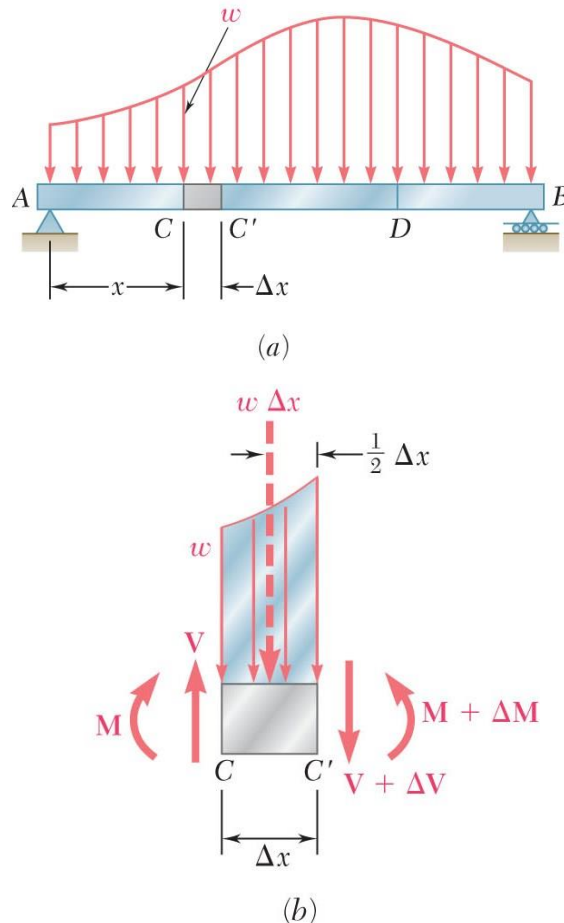


Fig. 5.9 (a) Simply supported beam subjected to a distributed load, with a small element between C and C', (b) Free-body diagram of the element.

- Relationship between load and shear:

$$\sum F_y = 0: V - (V + \Delta V) - w \Delta x = 0$$

$$\Delta V = -w \Delta x$$

$$\frac{dV}{dx} = -w$$

$$V_D - V_C = - \int_{x_C}^{x_D} w \, dx$$

$$= -(\text{area under load curve between } C \text{ and } D)$$

- Relationship between shear and bending moment:

$$\sum M_{C'} = 0: (M + \Delta M) - M - V \Delta x + w \Delta x \frac{\Delta x}{2} = 0$$

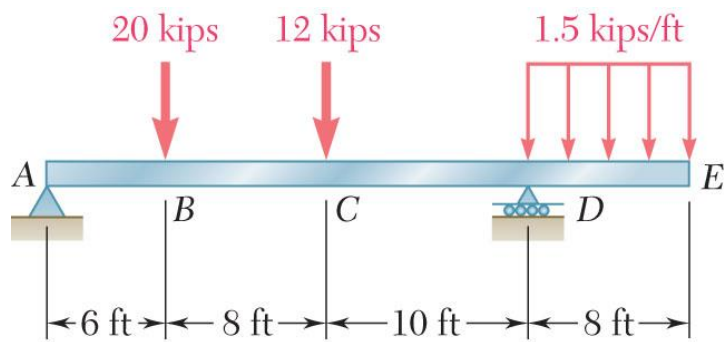
$$\Delta M = V \Delta x - \frac{1}{2} w (\Delta x)^2$$

$$\frac{dM}{dx} = V$$

$$M_D - M_C = \int_{x_C}^{x_D} V \, dx$$

$$= \text{area under shear curve between } C \text{ and } D$$

Sample Problem 5.3

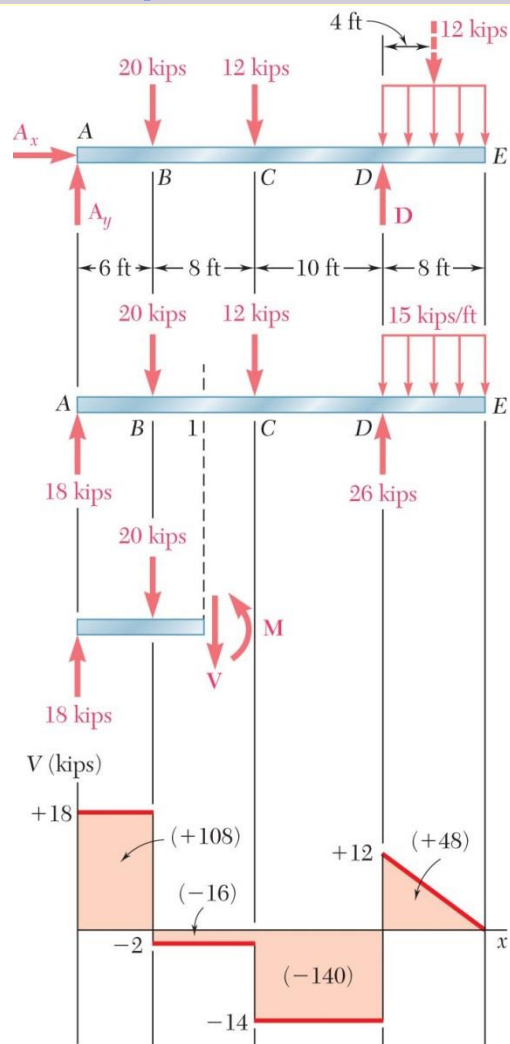


Draw the shear and bending moment diagrams for the beam and loading shown.

SOLUTION:

- Taking the entire beam as a free body, determine the reactions at A and D.
- Apply the relationship between shear and load to develop the shear diagram.
- Apply the relationship between bending moment and shear to develop the bending moment diagram.

Sample Problem 5.3



SOLUTION:

- Taking the entire beam as a free body, determine the reactions at A and D.

$$\sum M_A = 0$$

$$0 = D(24 \text{ ft}) - (20 \text{ kips})(6 \text{ ft}) - (12 \text{ kips})(14 \text{ ft}) - (12 \text{ kips})(28 \text{ ft})$$

$$D = 26 \text{ kips} \uparrow$$

$$\sum F_y = 0$$

$$0 = A_y - 20 \text{ kips} - 12 \text{ kips} + 26 \text{ kips} - 12 \text{ kips}$$

$$A_y = 18 \text{ kips} \uparrow$$

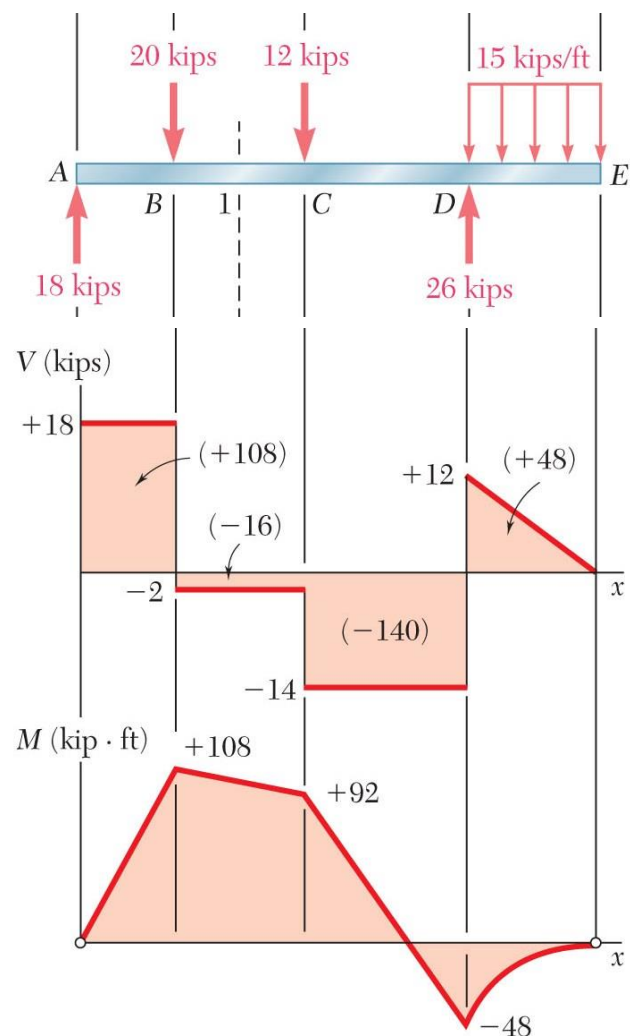
- Apply the relationship between shear and load to develop the shear diagram.

$$\frac{dV}{dx} = -w \quad dV = -w \, dx$$

- zero slope between concentrated loads
- linear variation over uniform load segment

Fig. 1 Free-body diagrams for finding the reactions as well as the shear diagram.

Sample Problem 5.3



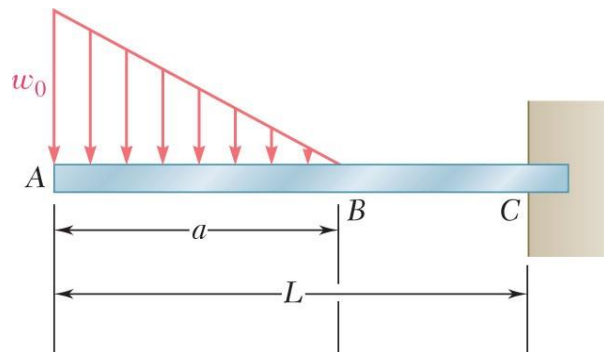
- Apply the relationship between bending moment and shear to develop the bending moment diagram.

$$\frac{dM}{dx} = V \quad dM = V dx$$

- bending moment at A and E is zero
- bending moment variation between A, B, C and D is linear
- bending moment variation between D and E is quadratic
- net change in bending moment is equal to areas under shear distribution segments
- total of all bending moment changes across the beam should be zero

Fig. 1 Free-body diagram for finding the reactions as well as the shear and bending-moment diagrams.

Sample Problem 5.5



SOLUTION:

- Taking the entire beam as a free body, determine the reactions at C.
- Apply the relationship between shear and load to develop the shear diagram.
- Apply the relationship between bending moment and shear to develop the bending moment diagram.

Draw the shear and bending moment diagrams for the beam and loading shown.

Sample Problem 5.5

SOLUTION:

- Taking the entire beam as a free body, determine the reactions at C .

$$\sum F_y = 0 = -\frac{1}{2} w_0 a + R_C \quad R_C = \frac{1}{2} w_0 a$$

$$\sum M_C = 0 = \frac{1}{2} w_0 a \left(L - \frac{a}{3} \right) + M_C \quad M_C = -\frac{1}{2} w_0 a \left(L - \frac{a}{3} \right)$$

Results from integration of the load and shear distributions should be equivalent.

- Apply the relationship between shear and load to develop the shear diagram.

$$V_B - V_A = -\int_0^a w_0 \left(1 - \frac{x}{a} \right) dx = -\left[w_0 \left(x - \frac{x^2}{2a} \right) \right]_0^a$$

$$V_B = -\frac{1}{2} w_0 a = -(\text{area under load curve})$$

- No change in shear between B and C .
- Compatible with free body analysis

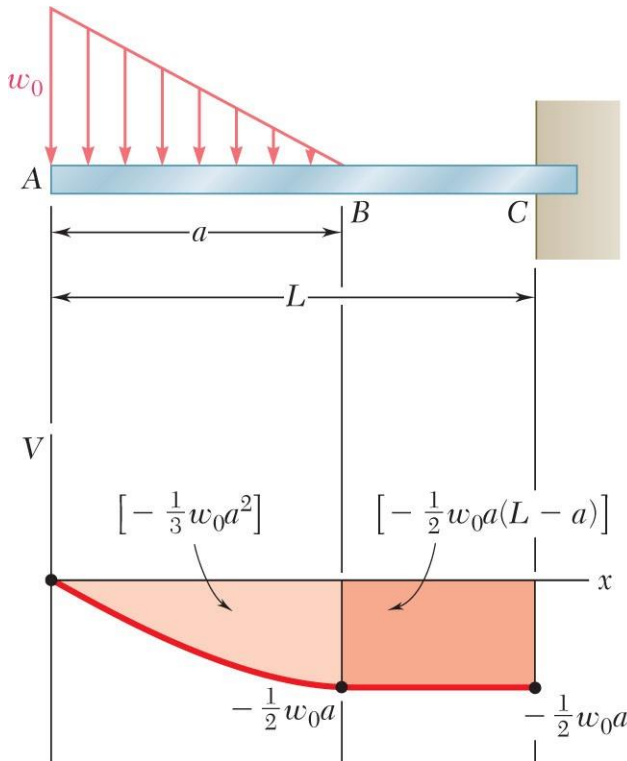
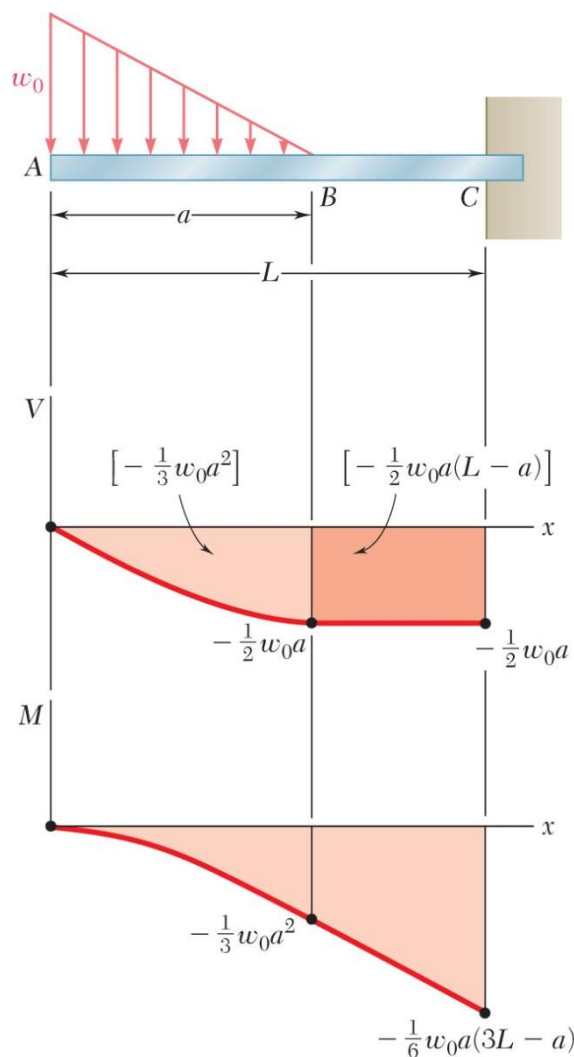


Fig. 1 Cantilevered beam having distributed loading and the resulting the shear diagram.

Sample Problem 5.5



- Apply the relationship between bending moment and shear to develop the bending moment diagram.

$$M_B - M_A = \int_0^a \left(-w_0 \left(x - \frac{x^2}{2a} \right) \right) dx = \left[-w_0 \left(\frac{x^2}{2} - \frac{x^3}{6a} \right) \right]_0^a$$

$$M_B = -\frac{1}{3} w_0 a^2$$

$$M_B - M_C = \int_a^L \left(-\frac{1}{2} w_0 a \right) dx = -\frac{1}{2} w_0 a (L - a)$$

$$M_C = -\frac{1}{6} w_0 a (3L - a) = \frac{a w_0}{2} \left(L - \frac{a}{3} \right)$$

Results at C are compatible with free-body analysis

Fig. 1 Cantilevered beam having distributed loading and the resulting the shear and bending-moment diagrams.