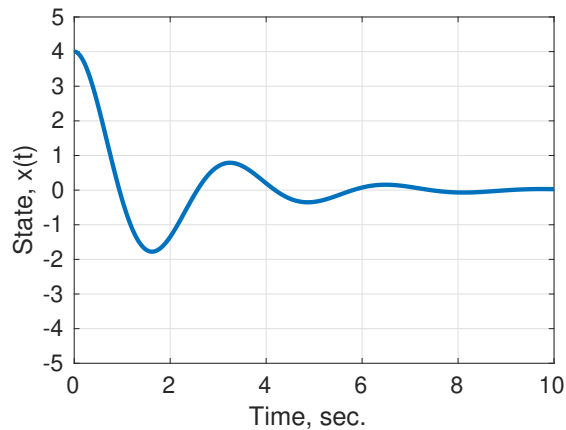


Name: \_\_\_\_\_

**MEGR 3122 Dynamics Systems II: Exam 2, Spring 2023**

*Directions: Circle the best answer. Show your work and explain your reasoning on all problems to receive full credit (unless otherwise specified).*

1. Consider the response of a homogeneous (unforced) second-order system from the initial condition  $x(0) = 4$  and  $\dot{x}(0) = 0$ .



Which ODE best matches the response shown?

- A.  $2\ddot{x} + 1\dot{x} - 0.5x = 0$
- B.  $4\ddot{x} + 4\dot{x} + 8x = 0$
- C.  $\ddot{x} + 2\dot{x} - 4x = 0$
- D.  $\ddot{x} + 8\dot{x} + 4x = 0$
- E.  $\ddot{x} + 2x = 0$

**Solution (B).** The response is underdamped so it cannot be E which has no damping. The response is also stable so it cannot be A or C which have negative spring coefficients. System D is already in standard form and has a natural frequency of  $\omega_n = 2$  and a damping ratio of

$$2\zeta\omega_n = 8 \implies \zeta = 8/(2\omega_n) = 2$$

Since this is an overdamped response one can conclude the solution must be B. This can be confirmed by normalizing the ODE to

$$\ddot{x} + \dot{x} + 4x = 0$$

which has a natural frequency of 2 rad/s. The damping ratio is

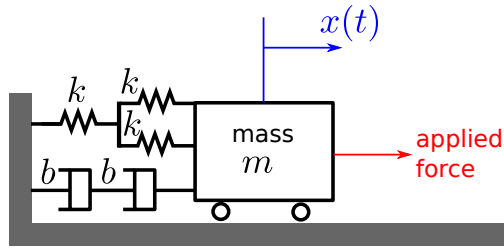
$$2\zeta\omega_n = 1 \implies \zeta = 1/(2\omega_n) = 0.25$$

which is underdamped as expected.

2. What is the damping ratio of the system shown below?

A.  $\frac{b}{\sqrt{(2/3)km}}$

B.  $\sqrt{3k/2m}$



C.  $\sqrt{1 - b^2} \sqrt{3k/m}$

D.  $\frac{b}{4\sqrt{(2/3)km}}$

E.  $\frac{3b^2}{2\sqrt{k/m}}$

**Solution (D).** The equivalent spring is

$$k_e = \frac{k(2k)}{k + 2k} = \frac{2k^2}{3k} = \frac{2}{3}k$$

and the equivalent damper is

$$b_e = \frac{b^2}{2b} = \frac{1}{2}b$$

The natural frequency is

$$\omega_n = \sqrt{\frac{k_e}{m}} = \sqrt{\frac{2k}{3m}}$$

and the damping ratio is

$$\Rightarrow \zeta = \frac{b_e}{2\sqrt{k_e m}} = \frac{(1/2)b}{2\sqrt{(2/3)km}} = \frac{b}{4\sqrt{(2/3)km}}$$

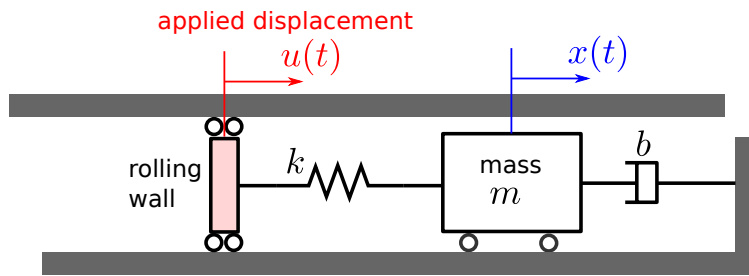
and

$$\zeta^2 = \frac{b^2}{16(2/3)km} = \frac{3b^2}{32km}$$

The damped natural frequency is then

$$\omega_d = \omega_n(1 - \zeta^2) = \sqrt{\frac{2k}{3m}} \left(1 - \frac{3b^2}{32km}\right)$$

3. Find the transfer function  $X(s)/U(s)$  for the mechanical system shown below



- A.  $\frac{k}{ms^2 + bs + k}$   
 B.  $\frac{1}{ms^2 - ks - b}$   
 C.  $\frac{k}{ms^2 + ks + b}$   
 D.  $\frac{b}{ms^2 + bs - k}$   
 E.  $\frac{b}{ms^2 + bs + k}$

**Solution (A).** Let  $i_1$  point to the right. The free body diagram on the mass has two forces (for the spring and damper) so that N2L is:

$$m\ddot{x} = -k(x - u) - b\dot{x}$$

Taking the Laplace transform

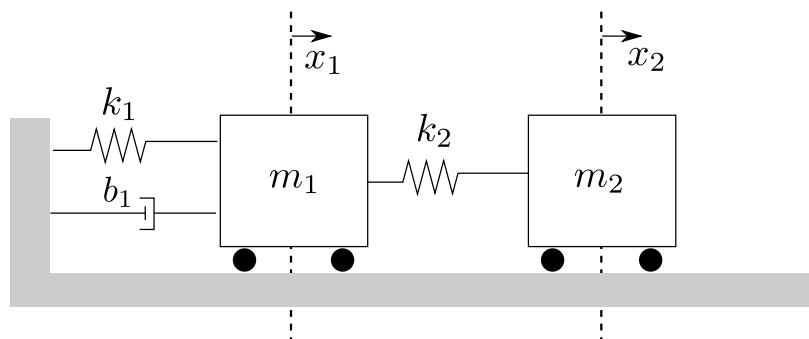
$$ms^2X(s) = -k(X(s) - U(s)) - bsX(s)$$

$$ms^2X(s) + kX(s) + bsX(s) = kU(s)$$

and rearranging

$$\frac{X(s)}{U(s)} = \frac{k}{ms^2 + bs + k}$$

4. Suppose that  $k_1 = 4$ ,  $k_2 = 10$ , and  $b_1 = 1$ . Both masses are equal  $m_1 = m_2 = 1$ . The first mass is at position  $x_1 = 0.5$  with velocity  $\dot{x}_1 = -1$  and the second mass is at position  $x_2 = 0.5$  with velocity  $\dot{x}_2 = 1$ . What is the total force that acts on the first mass at this instant? Assume that the positive directions for  $x_1$  and  $x_2$  are as indicated by the arrows below. Neglect gravity/normal forces in the vertical direction (consider horizontal forces only).

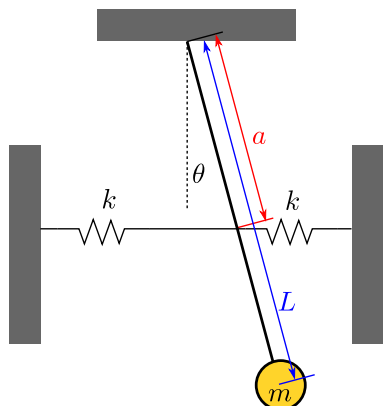


- A. The net force has magnitude 2 and points to the right
- B. The net force has magnitude 1 and points to the right
- C. The net force has magnitude 0
- D. The net force has magnitude 1 and points to the left
- E. The net force has magnitude 2 and points to the left

**Solution (D).** Since the two masses are displaced by the same amount there is no spring force due to spring  $k_2$ . The forces on mass one are

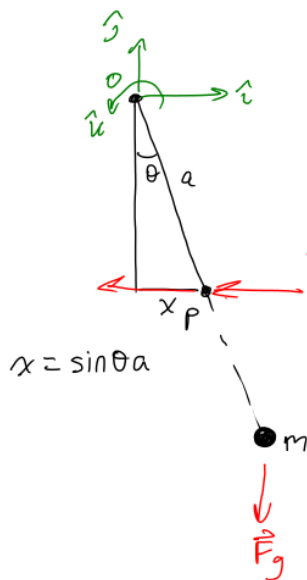
$$\sum F = -k_1 x_1 - b_1 \dot{x}_1 = -4(0.5) - 1(-1) = -2 + 1 = -1$$

5. The system below consists of a pendulum with a massless rod and bob of mass  $m$ . Two spring forces and gravity act on the system. What is the correct equation of motion? Assume small angles.



- A.  $\ddot{\theta} + \left( \frac{g}{L} + \frac{2ka^2}{mL^2} \right) \theta = 0$   
 B.  $\ddot{\theta} + \left( -\frac{g}{L} + \frac{2ka^2}{mL^2} \right) \theta = 0$   
 C.  $\ddot{\theta} + \frac{g}{L} \dot{\theta} + \left( \frac{2ka}{mL^2} \right) \theta = 0$   
 D.  $\ddot{\theta} + \frac{2ka}{mL^2} \dot{\theta} + \left( \frac{g}{L} \right) \theta = 0$   
 E.  $\ddot{\theta} + \left( -\frac{g}{L} + \frac{2ka}{mL^2} \right) \theta = 0$

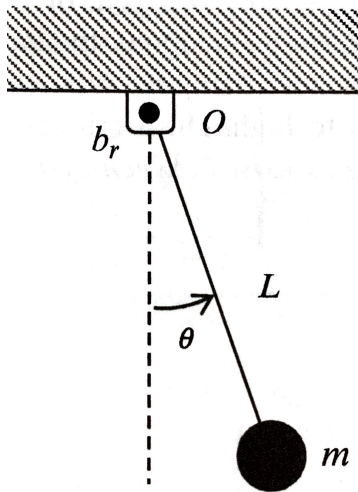
**Solution (A).** See below.



$$\begin{aligned}
 (mL^2) \ddot{\theta} &= \vec{r}_{p/o} \times (2\vec{F}_k) + \vec{r}_{m/o} \times \vec{F}_g \\
 &= -2(\cos \theta)a \cdot kx - mg \sin \theta L \\
 &= -2(\cos \theta)a \cdot k(\sin \theta a) - mg \sin \theta L \\
 &= -2ka^2 \cos \theta \sin \theta - mg \sin \theta L \\
 &\approx -2ka^2 \theta - mgL \theta
 \end{aligned}$$

$$\ddot{\theta} + \frac{2ka^2}{mL^2} \theta + \frac{mgL}{mL^2} \theta = 0$$

$$\ddot{\theta} + \left( \frac{2ka^2}{mL^2} + \frac{g}{L} \right) \theta = 0$$



6. Suppose that the highly underdamped system below has mass  $m = 3$  and length  $L = 2.5$ . It is released from rest and begins to oscillate (with very gradually decreasing amplitude due to the small damping  $b_r = 1$ ). What is the period of each oscillation in seconds? Assume small angles and  $g \approx 10$ .
- About two seconds
  - About half a second
  - About three seconds
  - About one second
  - About one tenth of a second

**Solution (C).** The equation of motion can be derived as

$$\sum M = -mgL \sin \theta - b_r \dot{\theta} = mL^2 \ddot{\theta}$$

which is re-written as

$$\ddot{\theta} + \frac{b_r}{mL^2} \dot{\theta} + \frac{mgL}{mL^2} \sin \theta = 0$$

Using small angles  $\sin \theta \approx \theta$  and simplifying

$$\ddot{\theta} + \frac{b_r}{mL^2} \dot{\theta} + \frac{g}{L} \theta = 0$$

The natural frequency is

$$\omega_n = \sqrt{\frac{g}{L}} = \sqrt{10/2.5} = 2 \text{ rad/s}$$

One period corresponds to  $2\pi$  radians and thus

$$T = 2\pi/\omega_n = 3.146$$

The damping of the system is negligible so the damped natural frequency is very close to the natural frequency.

7. Consider the multi-degree-of-freedom system of coupled ODEs with zero initial conditions:

$$\begin{aligned} \ddot{x}_1 + kx_1 - kx_2 &= 0 \\ \ddot{x}_2 + kx_2 &= f(t) \end{aligned}$$

where  $f(t)$  is an input into the system. What is the transfer function  $G_1(s) = X_1(s)/F(s)$ ?

- A.  $\frac{2k}{(s^2+k)^2}$
- B.  $\frac{k^2}{(s^2+k)}$
- C.  $\frac{k}{(s^2+k^2)^2}$
- D.  $\frac{k}{(s^2+k)^2}$
- E.  $\frac{3k}{(s^2+2ks+k)}$

**Solution (D).** Take the Laplace transform of each system

$$X_1(s)(s^2 + k) - kX_2(s) = 0 \quad (1)$$

$$X_2(s)(s^2 + k) = F(s) \quad (2)$$

Solving the first equation for  $X_2(s)$

$$X_2(s) = \frac{s^2 + k}{k} X_1(s)$$

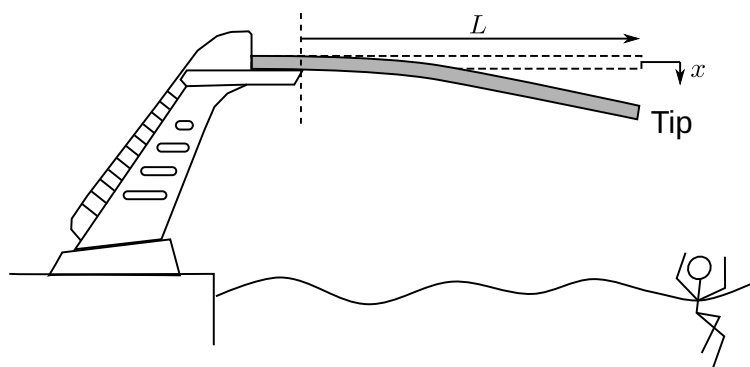
and plugging into the second equation

$$\frac{s^2 + k}{k} X_1(s)(s^2 + k) = F(s)$$

then rearranging

$$\frac{X_1(s)}{F(s)} = \frac{k}{(s^2 + k)^2}$$

8. A diving board has Young's modulus of  $E$ , width  $a$ , thickness  $b$ , area moment of inertia  $I$ , length of  $L$ , and (volumetric) density of  $\rho$ . Use a lumped-parameter model to model the displacement of the tip of the diving board. What is the natural frequency of the tip (in rad/s)?



- A.  $\frac{3}{4} \sqrt{\frac{0.23EI}{\rho abL^3}}$   
 B.  $\frac{3.61}{L^2} \sqrt{\frac{EI}{\rho ab}}$   
 C.  $\sqrt{\frac{0.23E(ab^2)}{4\rho L^3}}$   
 D.  $\frac{0.4796}{L^2} \sqrt{\frac{EI}{\rho ab}}$   
 E.  $\frac{L^2b}{0.277} \sqrt{\frac{\rho a}{EI}}$

**Solution (B).** The diving board is approximately a cantilever beam as shown in the lumped parameter tables. The total distributed mass of the beam is

$$m_d = \rho abL$$

thus the effective mass is

$$m_e = 0.23m_d = 0.23\rho abL$$

The effective spring constant is

$$k_e = \frac{3EI}{L^3}$$

The model of the equivalent system is

$$m_e \ddot{x} + k_e x = 0$$

and the natural frequency is

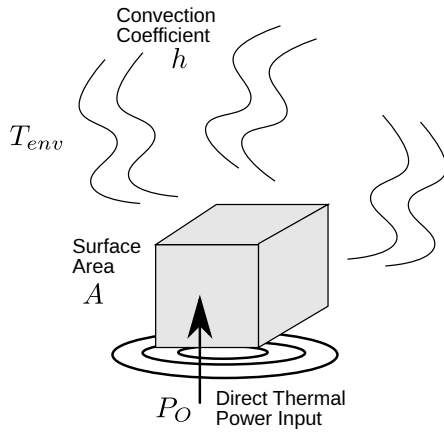
$$\omega_n = \sqrt{\frac{k_e}{m_e}} = \sqrt{\frac{3EI/L^3}{0.23\rho abL}} = \frac{3.61}{L^2} \sqrt{\frac{EI}{\rho ab}}$$

9. Suppose that an object initially at temperature  $T_0 = 65$  deg C has thermal mass of  $mc = 3$  J/(deg C) and is being simultaneously heated with a constant thermal power input of  $P_0 = 100$  Watts and cooled via convection with the environment at  $T_{\text{env}} = 15$  deg C. The exposed surface area for convection is  $A = 0.8$  m<sup>2</sup> and the convection coefficient is  $h = 2.5$  Watts/(m<sup>2</sup>· deg C). What will happen to the temperature of the object after a long period of time?

- A. The temperature will decay to a value  $T_{\text{env}} + P_0/(mc)$



- B. The temperature will approach an asymptote at  $(T_{\text{env}} + P_0/(mc))/2$
- C. The temperature will stay constant at  $T_0$
- D. The temperature will decay to  $T_{\text{env}}$  after approximately four time constants
- E. The temperature will oscillate between  $T_{\text{env}}$  and  $T_0 + P_0/(mc)$



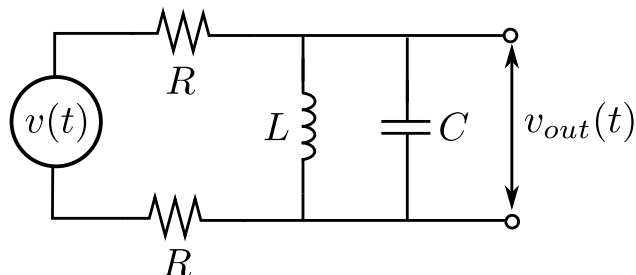
**Solution (C).** If we sum the thermal powers

$$\sum P = P_0 + hA(T_{\text{env}} - T) \quad (3)$$

$$= 100 + 0.8 \cdot 2.5(15 - 65) = 0 \quad (4)$$

thus the temperature will remain constant

10. For the following circuit, what is the transfer function  $G(s) = V_{\text{out}}(s)/V(s)$ ?



- A.  $\frac{CLs^2+1}{CLs^2+2RLs+1}$
- B.  $\frac{Ls}{2RLCs^2+Ls+2R}$
- C.  $\frac{1}{RLCs^2+2RLs+LC}$
- D.  $\frac{(1/RL)}{s^2+2Cs+1}$
- E.  $\frac{RCs^2+Ls+R}{CLs^2+LRs+1}$

**Solution (B).** The impedance of the inductor and capacitor in parallel is:

$$Z_{LC} = \left[ \frac{1}{Ls} + \frac{1}{1/Cs} \right]^{-1} = \left[ \frac{1}{Ls} + Cs \right]^{-1} = \left[ \frac{CLs^2 + 1}{Ls} \right]^{-1} = \frac{Ls}{CLs^2 + 1}$$

The impedance of the entire circuit is

$$Z_{\text{circuit}} = R + \frac{Ls}{CLs^2 + 1} + R = \frac{Ls + 2R(CLs^2 + 1)}{CLs^2 + 1} = \frac{2RLCs^2 + Ls + 2R}{CLs^2 + 1}$$

The desired transfer function is then

$$G(s) = \frac{Z_{LC}}{Z_{\text{circuit}}} = \frac{Ls}{2RLCs^2 + Ls + 2R}$$