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RBOT101: Mathematical Foundations of Robotics

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TABLE OF CONTENTS

LIST OF FIGURES	iv
LIST OF TABLES	v
CHAPTER 1 PREAMBLE	1
1.1 Course Description	1
1.2 Course Outcomes	1
1.3 Prerequisites	2
1.4 Recommended Texts	2
1.5 Recommended Journals	3
1.6 Required Software	3
1.7 Online Course Content	3
1.8 Assessments and Labs	4
1.9 Errata	4
CHAPTER 2 INTRODUCTION	5
2.1 Robots Components	6
2.2 Robot Spaces, Geometries, and Classifications	7
2.3 Characterization of Kinematic Geometry	8
2.3.1 Kinematic Geometries	9
2.3.2 Open Kinematic Chains	9
2.3.3 Closed-loop Kinematic Chains	12
2.4 Freedom and Structure in Mechanisms	13
2.4.1 Freedom, Connectivity and Mobility	13
2.4.2 Constraints and Freedoms	15
2.4.3 The Mobility Criterion	16
CHAPTER 3 MOTION OF RIGID BODIES	19
3.1 Screws	19
3.1.1 Motion screws: Twist's Pitch, Axis, and Magnitude	20
3.1.2 Dynamics Screws: Twist's Pitch, Axis, and Magnitude	22
3.1.3 Screw Motions	23

3.2	Rodrigues' Formula	23
3.3	The Matrix Exponential, The Lie Group and Lie Algebra	24
3.3.1	The Lie Group Notations	27
3.3.2	Exponential Map and Kinematic Chains	27
3.4	Rigid Body Transformations	28
3.4.1	Translation in \mathbb{R}^3	30
3.4.2	Rotations in \mathbb{R}^3	32
3.4.3	Rotation Matrices as Transformations	35
3.4.4	Planar Rotations	35
3.4.5	Composition of Rotations	38
3.5	Parameterization of Rotations $\in SO(3)$	42
3.5.1	Axis-Angle Parameterizations	42
3.5.2	Euler Angles	43
3.5.3	Quaternions	45
3.6	Homogeneous Coordinates	46
3.7	Rigid Body Velocities	47
3.7.1	Rotational Velocities	48
3.7.2	Rigid Body Velocity	49
	REFERENCES	51

LIST OF FIGURES

2.1	Early serial kinematic chain robot manipulators. Left: The Stanford Arm Serial Manipulator, 1969. ©Infolab, Stanford University. Right: The PUMA robot arm. Reprinted from Wikipedia.	5
2.2	Soft Robot Manipulators Left: An octopus inspired soft robot. Robots classified under muscular hydrostats. ©Cecilia Laschi. Right: A soft parallel robot manipulator (Hopkins et al., 2015).	6
2.3	Schematic description of in-series-actuated robot arms: (a) rotary joint actuated “about” the hinge (b) prismatic joint actuated “across” a hinge. Reprinted from (Hunt, 1983).	9
2.4	The Staubli 6-DOF Arm is an example of a Spherical Manipulator. Reprinted from DirectIndustry’s Webpage.	10
2.5	The SCARA Arm. ©Fanuc America.	11
2.6	Stewart-Gough Platforms Left: The 1954 Octahedral hexapod of the original Gough platform. Right: The retired platform from Dunlop tires in 2000. Picture courtesy of Parallelic.org.	12
2.7	The Lower Kinematic Pairs. Reprinted from (Hunt, 1977).	14
2.8	The planar RRR linkage, (left) is modified in (right) to allow spatial spin-movement of the coupler 3; the connectivity $\mathcal{C}_{13} = 2$). Reprinted from (Hunt, 1977).	14
2.9	Left: A Parallel Planar Robot Mechanism. Right: Workspace of the mechanism.	18
3.1	Illustration of rotation of interconnecting link of two spherical links.	25
3.2	Lie Group and their Notations. Reprinted with permission from Federico Renda. IROS 2018, Screw Theory Tutorial.	28
3.3	An illustration of the exponential map for a multi-body rigid system. Reprinted with permission from Federico Renda. IROS 2018, Screw Theory Tutorial. . . .	29
3.4	A point q in space with respect to two coordinate frames.	31
3.5	An illustration of the relative orientation of a rigid object q between an inertial frame I and a body frame J	33
3.6	Illustration of rotation between two frames in a plane.	36
3.7	Relative orientation between two frames.	38
3.8	Coordinate frames on a body. Reprinted from (Spong et al., 2006)	39
3.9	Illustration of composition of rotations about a current axis.	41
3.10	An illustration of the ZYZ rotation angles	43

LIST OF TABLES

CHAPTER 1

PREAMBLE

Consider this the roadmap for this course. Please read through the syllabus posted on Moodle2 carefully and feel free to share any questions that you may have. Please print a copy of the Syllabus for reference. Some relevant parts of the Syllabus are repeated here but the Moodles reference should serve as your guide throughout the ten weeks of this course.

1.1 Course Description

This course focuses on the algorithmic and mathematical concepts with respect classical and recent methods for solving real-world problems in robotics. While some students may have encountered some of the concepts we will be treating in past courses or avenues of study, we will provide the breadth and depth necessary for equipping students to be world-class roboticists. The topics covered by this course shall include the configuration space, rigid bodies, semi-rigid soft bodies, as well as their motions in \mathbb{R}^n , wrenches, homogeneous transformations, optimal algorithms for rigid body rotations, linear systems theory, probability theory, the Kalman filter. The course will begin and end with a self-assessment to allow students to gauge their strengths and weaknesses in these topics. References for further, in-depth study in each topic are provided at the end of this course.

1.2 Course Outcomes

After taking this course, each student will be able to

- Develop mathematical tools for solving fundamental kinematic problems in robot operation;
- Formulate optimal state estimation tools for solving real-time smoothing and filtering operations in robotics;

- Integrate state estimation with rigid and semi-rigid soft bodies to solve real-world automation problems; and 4. Use open-source Python, and C++ tools to solve classical and emerging problems in robotics in our day.

1.3 Prerequisites

An undergraduate-level understanding of linear algebra, analytical mechanics, Python and C++ programming.

1.4 Recommended Texts

- Main Texts
 - Simon, Dan. (2007). Optimal state estimation: Kalman, $H-\infty$, and nonlinear approaches. Choice Reviews Online, Vol. 44, pp. 44-3334-44-3334. <https://doi.org/10.5860/ch3334>
 - Murray, R. M., Li, Z., and Sastry, S. S. (1994). A Mathematical Introduction to Robotic Manipulation. Book (Vol. 29). Free PDF preprint downloadable from, [Murray's website](#).
 - Theory of Screws: A Study in the Dynamics of a Rigid Body by Robert Stawell Ball, Dublin: Hodges, Foster, and Co., Grafton-Street. a. Textbooks:
- Secondary Text
 - Modern Robotics: Mechanics, Planning, and Control. Free PDF preprint downloadable from [Author's Northwestern University Website](#).
- Auxiliary Text:

- Theory of Screws: A Study in the Dynamics of a Rigid Body by Robert Stawell Ball, Dublin: Hodges, Foster, and Co., Grafton-Street (Should be downloadable via Interlibrary Loan).

1.5 Recommended Journals

- [IEEE Transactions on Robotics](#).
- [The International Journal of Robotics Research](#).
- [The IEEE International Conference on Robotics and Automation \(ICRA\)](#).
- [IEEE/Robotics Society of Japan International Conference on Intelligent Robots and Systems \(IROS\)](#).
- [Robotics and Autonomous Systems, An Elsevier Journal](#).

1.6 Required Software

- A working knowledge of python and the anaconda environment.
- ROS 1.x Installation Instructions: [ros 1.x website](#).
- ROS 2 installation [ros 2.0 website](#).

1.7 Online Course Content

This course will be conducted completely online using Brandeis' LATTE [site](#). The site contains the course syllabus, assignments, our discussion forums, links/resources to course-related professional organizations and sites, and weekly checklists, objectives, outcomes, topic notes, self-tests, and discussion questions. Access information is emailed to enrolled participants before the start of the course. To begin participating in the course, review the “Welcoming Message” and the “Week 1 Checklist.”

1.8 Assessments and Labs

Please read the syllabus posted on the RBOT 101 website thoroughly.

1.9 Errata

If in the course of using these notes, you find sentence errors, errata or mistakes in equations, please annotate them and upload it to the discussion forum. Points will awarded, at the discretion of the instructor, for such help.

CHAPTER 2

INTRODUCTION

We are chiefly concerned with rigid bodies and occasionally semi-rigid or soft bodies connected together by joints. In general, we take robots to be mechanisms that are made up of links connected to one another by joints. Typically, the joints connect two or more links and are formed by simple contact with adjacent bodies. Sometimes, the joints may be flexible – whether by belt, band, spring or some kind of elastic component such as bellows, diaphragms, tendons ([Bern et al., 2017](#)), fiber-reinforced elastomers ([Bishop-Moser et al., 2012](#)), resilient pads, strip, or bush. The assembly formed after the various connections between links and joints are called a kinematic chain.

A kinematic chain is a form of a mechanism. A “mechanism can be interpreted as a means of transmitting, controlling, or constraining relative movement” ([Hunt, 1977](#)). The term kinematics is generally used to describe the motion of a rigid body in space. It also applies to the motion of a semi-rigid or a completely soft robot in space. The rigid, semi-rigid or completely soft kinematic systems that allow a body to exhibit motion under a/some

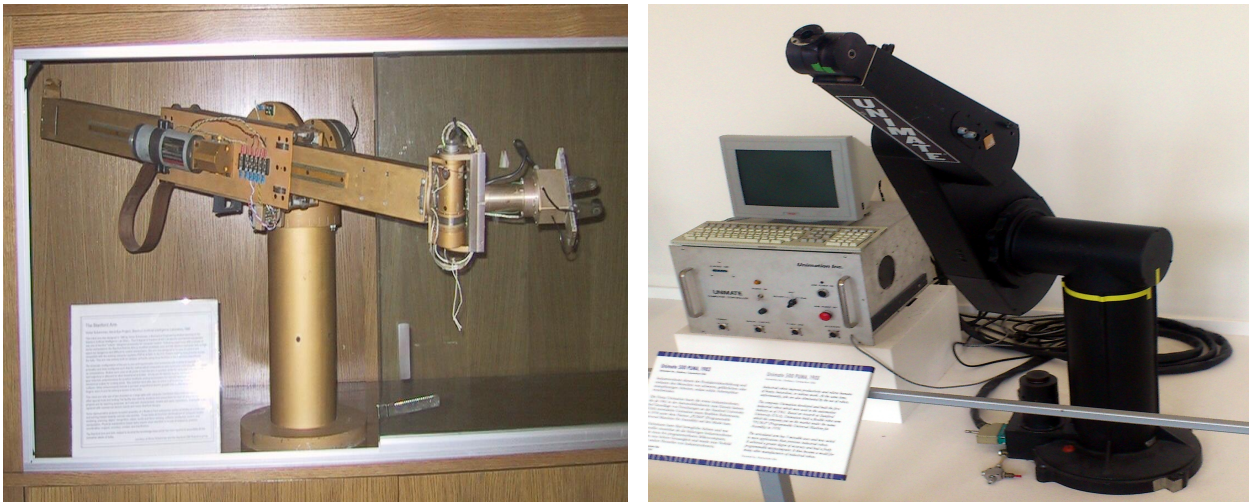


Figure 2.1: Early serial kinematic chain robot manipulators. Left: The Stanford Arm Serial Manipulator, 1969. ©Infolab, Stanford University. Right: The PUMA robot arm. Reprinted from Wikipedia.

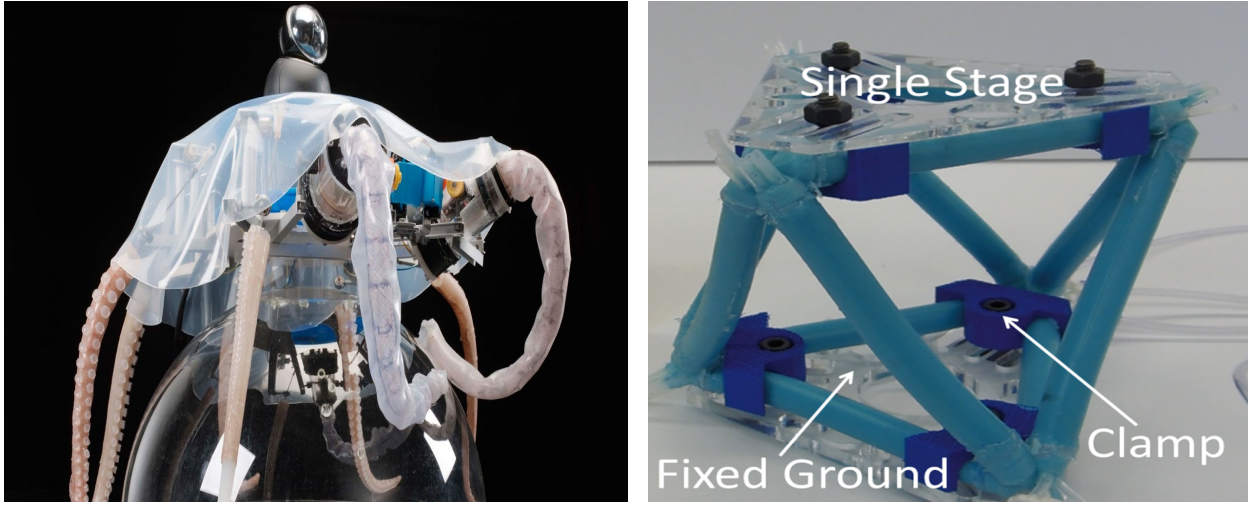


Figure 2.2: Soft Robot Manipulators Left: An octopus inspired soft robot. Robots classified under muscular hydrostats. ©Cecilia Laschi. Right: A soft parallel robot manipulator (Hopkins et al., 2015).

controlled motion of its freedom in space with respect to a fixed base frame are what we describe in this module.

2.1 Robots Components

The joints of a robot can be revolute or prismatic. Revolute joints allow rotation about a fixed axis between two or more links while prismatic joints allow a linear motion about an axis – which is between two links. Symbolically, the revolute joint is denoted by R while the prismatic joint is denoted by P . By this logic, it follows that a robot made up of three links that are connected by a prismatic joint would be characterized as a PPP arm, while a three link arm whose links are all connected by three rotary joints would be denoted as an RRR arm.

A mathematical way to abstract the interconnection between the links of a robot arm is to denote the axis of rotation or the axis of translation by z_i for the joint that connects link i to link $i + 1$. When we characterize the robot's motion, it often helps to work in a so-called generalized coordinate. Now consider a rigid body such as a link of a robot arm for

example. This link contains myriads of particles, with each particle having its own position and orientation in the world. How can we describe the motion of this rod given its infinitely many particulate matter? For rigid bodies, classical mechanics comes to the rescue. As such, for a revolute joint of a rigid robot manipulator for example, the generalized coordinate is often the joint angle θ , while for a prismatic joint it is the link length d – representing the relative displacement between adjacent links. When the body we speak of is a continuum, the motion of each particle within a link needs to be accounted for. In such scenarios, researchers often work with a so-called a [Frenet-Serret](#) frame that models a curve on the soft robot’s surface – with or without torsion. This simplifies the dynamics and kinematics of the system and the specification of these generalized coordinates uniquely determines the position of all the particles (in a material sense for a rigid system) that make up the robot. For more examples on parameterizing soft robots, please look through some of the following references: ([Ogunmolu et al., 2015](#)), ([Hannan and Walker, 2000](#)) ([Ogunmolu et al., 2016](#)), ([Renda and Seneviratne, 2018](#)), ([Shepherd et al., 2011](#)), ([Hannan and Walker, 2003](#)), ([Ogunmolu et al., 2017](#)), ([Giorelli et al., 2015](#)), ([Ogunmolu et al., 2020](#)), ([Renda et al., 2014](#)), ([Ogunmolu, 2019a](#)), ([Ogunmolu, 2019b](#)).

2.2 Robot Spaces, Geometries, and Classifications

The location of a robot in the world is described by its configuration. This configuration specifies the positions of all points of the robot. While the coordinates of these points may be multiple over a continuous range of real numbers, the smallest number of independent variables of these real-valued coordinates needed to represent the robot configuration are the degrees of freedom of the system. In general, a free rigid body has six degrees of freedom. The motion of the robot could be about a translation or rotary axis. The n -dimensional space that contains all the possible configurations of the robot is the configuration space (or C-space) of the robot. The configuration of the robot is expressed by a point in its C-space.

A manipulator with more than 6-DOF is said to be kinematically redundant. The set of variables that together with a description of the manipulator's dynamics and future inputs that determines the future transient response of the manipulator is the state of the robot, while the set of all possible states is referred to as the state space. For a rigid robot manipulator for example, the material and referential description alone is sufficient to describe the dynamics, which are Newtonian in nature. For such systems, the state may be specified by the joint variables $\{q, \dot{q}\}$ signifying the joint angles and joint velocities. For continuum systems, such representations are often inadequate. A popular method is to introduce a Frenet-Serret (you can read more about F-S frames in the link included earlier) frame on the body of the soft robot (SoRo) and characterize the state space based on the three parameters *i.e.* $\{\mathcal{K}, l, \alpha\}$ that characterize the kinematic motion of the body viz., curvature of an arc projected on the SoRo's body, \mathcal{K} , the arc's length, l , and the angle subtended by a tangent along that arc, α .

2.3 Characterization of Kinematic Geometry

As mentioned earlier, disassociated from the dynamics (*i.e.* forces, torques and such) of a body, that which deals with the displacements, or movements of a body relative to another within a mechanism is termed the kinematics. The displacement could be linear, angular – and in combination with the derivatives with respect to time of such displacements viz., velocities, accelerations, and hyper-accelerations all are part of the kinematics of a body. The other branch of dynamics, called kinetics, determines the forces, torques, energy, momenta, inertia, equilibrium and dynamic stability of the system and that is treated in a separate module in this course.

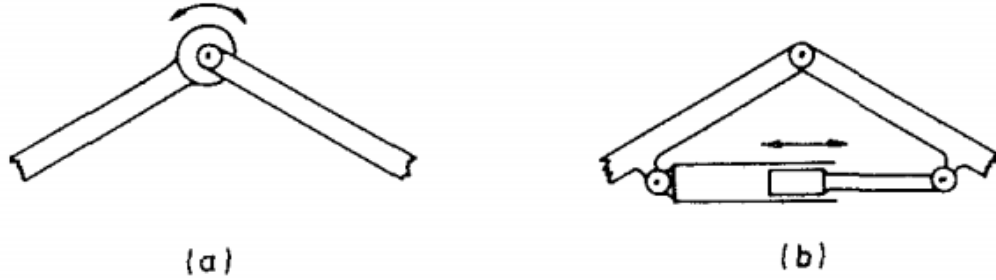


Figure 2.3: Schematic description of in-series-actuated robot arms: (a) rotary joint actuated “about” the hinge (b) prismatic joint actuated “across” a hinge. Reprinted from (Hunt, 1983).

2.3.1 Kinematic Geometries

Kinematic geometry deals only with the displacements of a system in the first and simplest segment of kinematics. In general, the use of time as a variable is shunned upon since the displacements we intend to carry out may be performed as fast as user wishes in their implementation. For the rest of this subsection, by kinematics, we shall mean the displacement of rigid bodies or “the solid geometry of relatively moving bodies”.

The majority of robot arms (at least today) have their actuators (or joints) connected in series along an anthropomorphic arm, with each joint being either at or associated with a single DOF in the robot arm. Fig. 2.3 describes the two major ways rigid links are actuated today. From a geometric point of view, both types of actuation in Fig. 2.3 serve the same purpose, which is to control the rotation of the joint. However, when we are interested in controlling “pure translation” such as sliding between links, non-pivoted linear actuators are used on their own.

2.3.2 Open Kinematic Chains

When the links are serially connected to one another and there is no link that connects the base link to the tool frame, the robot is said to be in-series actuated. Series-jointed



Figure 2.4: The Staubli 6-DOF Arm is an example of a Spherical Manipulator. Reprinted from DirectIndustry's Webpage.

links have a major disadvantage, namely that they accumulate errors from shoulder out to end-effector and they suffer from a lack of rigidity. In order to eliminate their inherent load-dependent error, manufacturers typically stiffen them. However, this stiffening process increases the mass of the arm such that greater demand is placed on the actuation system. Some manufacturers employ a compensated actuation or sophisticated control techniques in order to overcome the load-dependent errors.

Examples of in-series actuated mechanisms is the spherical robot (see [Fig. 2.5](#)), where a succession of the robot segments are linked to the predecessors by revolute joints. By actuating each of the n joints we can control the n degrees of freedom of the end-effector. The SCARA robot, shown in [Fig. 2.5](#) is an example that allows the control of the end-effector based on the geometry of the previously connected links.

It is ideally desirable to keep the load-to-mass ratio of a robot as minimum as possible whilst preserving positioning accuracy. This is so as to preserve two important properties of

the robot: repeatability (*i.e.* the maximum distance between two positions of the end-effector reached for the same desired pose from different starting positions), and absolute accuracy (*i.e.* distance between the desired and actual position of the end-effector) respectively. Positioning accuracy is a function of the deformation of flexure – typically not accounted for by the robot’s internal sensors – and the absolute accuracy. The absolute accuracy is a function of the sensors at the manipulator joints, the clearance for the drive, flexure of links, and geometric accuracy of link orientations *inter alia*. When geometric constraints are violated (take a small error in the perpendicularity between successive links of a spherical arm, for example), large errors are bound to occur in the vertical motions so that such errors must be accounted for during manufacture. The conventional approach to overcoming this currently is for manufacturers to stiffen the robot’s links. However, stiffening the links is tantamount to overall heavier mass, which in turn means the manipulator experiences higher inertia, centrifugal and Coriolis forces – these complicate motion planning for complex trajectories.

As we close this part of the module, bear in mind that open kinematic chains are governed by inertia and centrifugal forces – forces that exist on different scales. For example, inertia forces are a function of the square of the link lengths while frictional forces are not affected by the such dimensions. As such, serial robots cannot be scaled down to a micro level as inertia



Figure 2.5: The SCARA Arm. ©Fanuc America.

forces would be reduced while frictional forces would remain unchanged. It is not surprising that these attributes listed make serial manipulators exhibit poor positioning accuracy.

2.3.3 Closed-loop Kinematic Chains

When the number of links connected to a joint (the connection degree) is more than 2, we have a closed-loop kinematic chain. Closed-loop kinematic chains resolve the accuracy problems of open-loop chains by mechanically distributing the load on the links: they link the end-effector to the ground by a set of links that support only a fraction of the load. While theoretical works that envisioned parallel mechanisms have been in existence since 1645 (Merlet, 2015), the Stewart (Stewart, 1965) and Gough (Gough, 1957) platforms are the original designs of closed-loop chains on record. The mechanical arrangement of the links



Figure 2.6: Stewart-Gough Platforms Left: The 1954 Octahedral hexapod of the original Gough platform. Right: The retired platform from Dunlop tires in 2000. Picture courtesy of Parallemic.org.

of the platform help with the load-to-mass ratio: the manipulator mass is reduced, and the disturbing effects of the Coriolis forces decreases. Albeit, these manipulators typically have a smaller workspace. Formally, we define a generalized parallel manipulator as a closed-loop

kinematic chain mechanism whose end-effector is linked to the base by many independent kinematic chains (Merlet, 2015).

2.4 Freedom and Structure in Mechanisms

Early on, we briefly defined what constitutes the degrees of freedom of a rigid body. We will now systematically analyze how to determine the mobility of a mechanism given its kinematic pairs, linkages and freedoms.

2.4.1 Freedom, Connectivity and Mobility

For every kinematic pair¹, there exists a characteristic number of the degrees of freedom that characterize its mobility. If a kinematic pair has elements that touch at a single point, we would have five degrees of freedoms – two of which would be translatory and three rotary. A kinematic pair whose elements always touch along a line or a curve has four or fewer degrees of freedom, or for short, freedom (see Fig. 2.7).

However, in mechanisms, we are more concerned about the relative motion between objects in the mechanisms that do not directly touch one the other. In such systems, what we do is consider the freedom that is contributed by each of the several kinematic pairs that connect them. For the popular four bar linkage, for example (see (Murray, 1994)), the four R -pairs on the two connectivity-links 2 and 4 complete a coupling or connection between 1 and 3. This is said to have a connectivity of 1, typically written as $\mathcal{C}_{13} = 1 = \mathcal{C}_{31}$. Note that here, all connectivities are the same, and $\mathcal{C}_{ij} = 1$, for $i \neq j$ and $i, j = 1, 2, 3, 4$. Now, suppose that the R pairs are replaced by spherical, or for short, S pairs, then we have an additional so-called spin-freedom about the SS axis (see figure Fig. 2.8) so that $\mathcal{C}_{13} = 2$. In any case, $\mathcal{C}_{12} = \mathcal{C}_{13} = \mathcal{C}_{43} = 2$.

¹Bonus homework: Research what is a kinematic pair and produce a one-page written report.

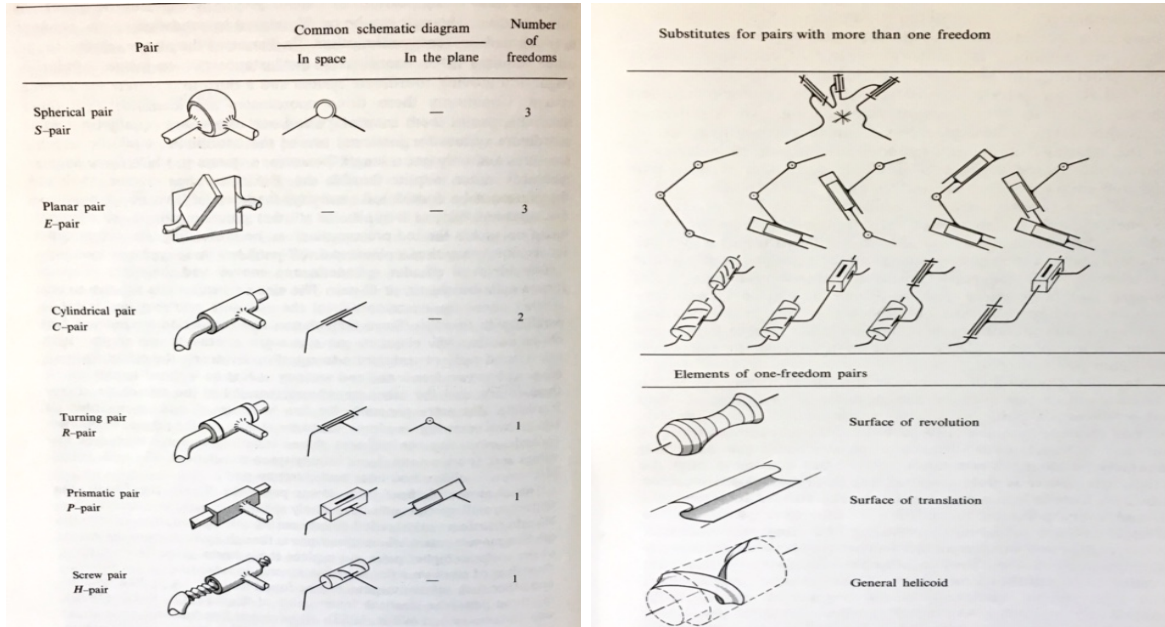


Figure 2.7: The Lower Kinematic Pairs. Reprinted from (Hunt, 1977).

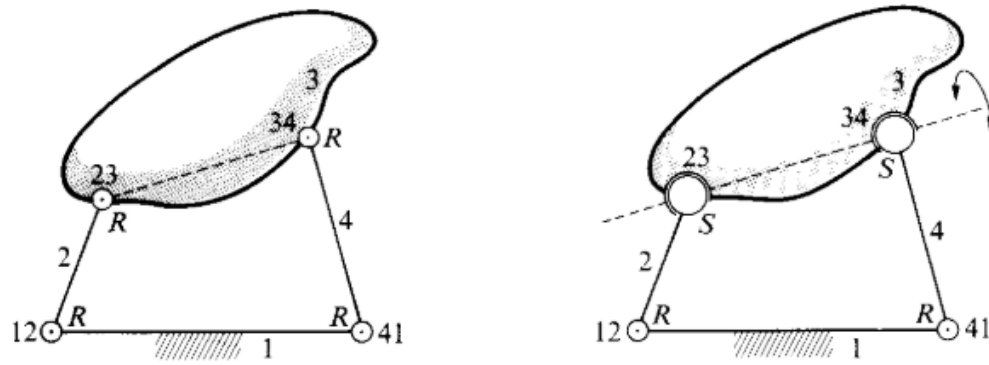


Figure 2.8: The planar RRR linkage, (left) is modified in (right) to allow spatial spin-movement of the coupler 3; the connectivity $\mathcal{C}_{13} = 2$). Reprinted from (Hunt, 1977).

Related to the concept of connectivity and more popular in literature is the concept of relative mobility or simply mobility. The mobility, \mathfrak{M} , is usually referred to as the number of degrees of freedom of a mechanism. To specify the independent variables needed to determine all the relative locations of the complete members of a mechanism with respect to one another, we typically use the mobility criterion. As such, we would have the kinematic chain of the left inset of Fig. 2.8 having $\mathfrak{M} = 1$ since only an angle between the elements of

any of the four kinematic pairs is required so as to prevent all relative movement. However, for the *RSSR* mechanism on the right of Fig. 2.8, the mobility would be $\mathfrak{M} = 2$ since member 3 has a spin-freedom.

Determining Degrees of Freedom

Screw Coordinates are better suited – from a kinematic standpoint, at any rate – to determine the general and impartial location of a rigid body.

2.4.2 Constraints and Freedoms

In rigid body kinematics, it is a given that the freedoms of a free rigid body is 6. But how is this determined? Suppose we have six homogeneous screw coordinates (to be introduced shortly) x_1, x_2, \dots, x_6 , we find that the body is fixed when values are attached to all six of them. However, the physical constraining system to which the body is subjected is not likely to have for every constraint exactly one corresponding independent screw coordinate. For every constraint in the system, there is an influence on each of the six coordinates. For every independent constraint, a freedom of the body is suppressed so that the six independent constraint-equations $f_i(x_1, x_2, \dots, x_6) = 0$, $i = 1, 2, \dots, 6$ must all be simultaneously satisfied. We define the condition as the constraint equation that describes a particular constraint algebraically. As the constraints are progressively relaxed, the corresponding constraint-equations are struck out one after another, so that the body acquires one, two, \dots , degrees of freedom, until the body is completely free with no constraints or equations and we end up with six freedoms. Suppose the number of freedoms is f and the number of unfreedoms or constraints is u , it turns out that

$$u + f = 6. \tag{2.4.1}$$

2.4.3 The Mobility Criterion

For g working joints between a total of n bodies, the number of relative degrees of freedom can be identified with the relative mobility of the system of bodies, described as

$$\mathfrak{M} = 6(n - 1) - \sum_{i=1}^g u_i \quad (2.4.2)$$

where the summation term aggregates over all individual unfreedoms. Plugging (2.4.1) into (2.4.2), we have

$$\mathfrak{M} = 6(n - 1) - \sum_{i=1}^g 6 + \sum_{i=1}^g f_i \quad (2.4.3)$$

or

$$\mathfrak{M} = 6(n - g - 1) + \sum_{i=1}^g f_i. \quad (2.4.4)$$

Equation (2.4.4) is termed the general mobility criterion. It is attributed to Grübler (1908 and 1917), and independently to Kutzbach (1929). In this module, we will generally call it the Grübler-Kutzbach's mobility criterion. When there are independent kinematic chains within the body of concern, it is sometimes more convenient to write (2.4.4) as

$$\mathfrak{M} = \sum_{i=1}^g f_i - 6c \quad (2.4.5)$$

with c being the number of independent chains.

There are exceptions to the Grübler-Kutzbach's mobility criterion such as when we have planar or spherical mechanisms. Take the four-bar linkage for example. There are four freedoms to the kinematic pairs, yet patently it has a mobility of $\mathfrak{M} = 1$, and not $\mathfrak{M} = -2$ as (2.4.5) would have us so determine it. This is because for planar and spherical mechanisms, the number of freedoms is not six as in general space, but three. Therefore, we write out the special version of (2.4.4) as

$$\mathfrak{M} = 3(n - g - 1) + \sum_{i=1}^g f_i. \quad (2.4.6)$$

This special nature of (2.4.6) arises because the joints' freedoms are not independent in planar motion, because the axes of the turning pairs are all directed parallel to one another, any prismatic pair being perpendicular to them. For spherical motions, the turning axes co-intersect at a single point.

Homework 1. Read up on the common kinematic arrangements in Section 1.3 of (Spong et al., 2006) and produce a 2-page single-spaced summary report. Your report must not contain diagrams but feel free to do as much analysis of the configurations of the various kinematic arrangements that are mentioned.

- Using the mobility condition, determine and explain why the SCARA robot of Fig. 2.5 has the number degrees of freedom that you find.
- For the mobile manipulator we are using in this course, analyze the connectivities and freedoms of the kinematic pairs in the mechanism. In addition, determine the freedom of the overall mechanism and write out the mobility criterion.
- With the Grübler-Kutzbach's mobility condition that we have learned, analyze the mobility criteria of the mechanism of Fig. 2.9. Hint: This mechanism is made up of two chains: chains $A_3 B_3 B_1 A_1$ and $A_2 B_2 B_4 A_4$, and there is a fixed distance between the U -joints, $A_2 A_3$ as well as A_1, A_4 .

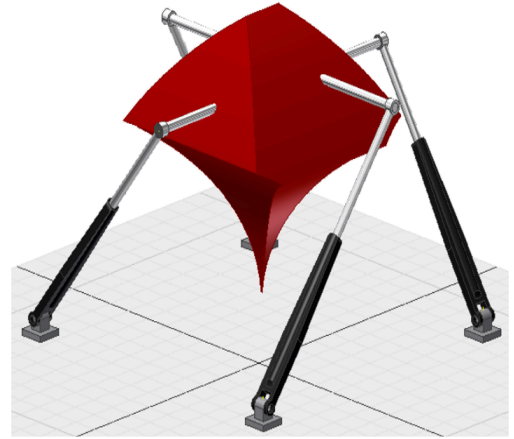
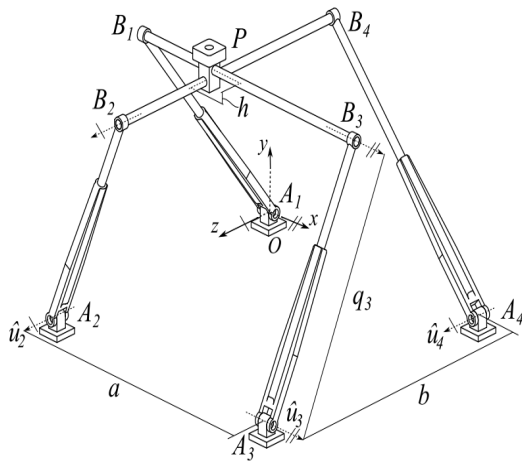


Figure 2.9: Left: A Parallel Planar Robot Mechanism. Right: Workspace of the mechanism.

CHAPTER 3

MOTION OF RIGID BODIES

In this chapter, we shall lay the foundations for analyzing the kinematics of a rigid or soft body in general space. Our goal is to present a geometric view of the translational and rotational rigid body motions. We take the classical screw theory approach owing to its simplicity in analyzing the geometry of kinematic motions compared against the common Denavit-Hartenberg (DH) conventions. We leave the treatment of the DH convention as an exercise for the reader. While the materials presented here may appear abstract, the applications are practical and useful. Therefore in reading the materials presented forthwith, it is recommended to the reader to ask what is it to the kinematic geometry of mechanisms that we have labored upon in [chapter 1](#). The recommended texts for this chapter are

- Murray, R. M., Li, Z., & Sastry, S. S. (1994). A Mathematical Introduction to Robotic Manipulation . Book (Vol. 29), Chapter 2
- A treatise on the theory of screws, Sir R.S. Ball, Chapter 1
- Screw theory for robotics, Jose M. Pardos-Gotors, IROS2018 tutorial; (Email the instructor for a copy).

3.1 Screws

Simply put, a screw is a line (called an axis) by which a definite linear magnitude (termed the pitch, p) is associated. A screw may be defined by a 6-vector of screw coordinates, $\underline{s} = (S_1, S_2, S_3, S_4, S_5, S_6)$ which has the following interpretations in terms of the Plücker line coordinates of the axis

$$L = S_1, \quad M = S_2, \quad N = S_3, \tag{3.1.1a}$$

$$P = S_4 - pS_1, \quad Q = S_5 - pS_2, \quad R = S_6 - pS_3 \tag{3.1.1b}$$

where L, M, N are proportional to the direction cosines of the line forming the axis while P, Q, R are proportional to the moment of the line about the origin of the reference frame. If we scale the Plucker coordinates by $(L^2 + M^2 + N^2)^{1/2}$, the first three would represent the direction cosines of the line while the last three would represent the moments. We have the moment of the line as the cross product of a vector from the reference frame's origin to any point on the line with the unit vector in the line direction. Thus, we have the pitch of the screw defined as

$$p = \frac{S_1 S_4 + S_2 S_5 + S_3 S_6}{S_1^2 + S_2^2 + S_3^2} \quad (3.1.2)$$

whose magnitude is

$$\|\ell\|^2 = S_1^2 + S_2^2 + S_3^2 \quad (3.1.3)$$

for a finite pitch, otherwise, it is

$$\|\ell\|^2 = S_4^2 + S_5^2 + S_6^2 \quad (3.1.4)$$

for an infinite pitch. Note that for infinitesimal screws, scalar multiplication and vector addition are valid so that two screws, \underline{s}_1 and \underline{s}_2 are considered linearly dependent if there exists non-zero scalars, c_1 and c_2 *i.e.* $c_1 \underline{s}_1 + c_2 \underline{s}_2 = 0$.

3.1.1 Motion screws: Twist's Pitch, Axis, and Magnitude

The unique line about which a body in space rotates or translates for an infinitesimal motion or velocity of the body is called the twist axis. Formally, we say a body receives a twist about a screw when it is uniformly rotated about the screw, while it is translated uniformly parallel to the screw, through a distance equal to the product of the pitch and the circular measure of the angle of rotation (Ball, 1908).

Similar to the screw, the twist is characterized by a six-vector of coordinates, $\underline{t} = (T_1, T_2, T_3, T_4, T_5, T_6)$ so that the components of the angular velocity of the body are denoted by $\underline{w} = (T_1, T_2, T_3)$ while the components of the linear velocity of a fixed point in the body lying at the origin of the coordinate system is $\underline{v} \equiv (T_4, T_5, T_6)$.

An important thing to note is that the a twist is characterized by the attributes pitch, magnitude and an axis. Hence, we have the Plücker coordinates of the twist axis given as

Definition 1 (Twist Axis).

$$L = T_1, \quad M = T_2, \quad N = T_3 \quad (3.1.5a)$$

$$P = T_4 - pT_1, \quad Q = T_5 - pT_2, \quad R = T_6 - pT_3. \quad (3.1.5b)$$

Definition 2 (Pitch of a Twist). The pitch of the twist, $\xi = \begin{pmatrix} \omega & v \end{pmatrix}^T$ is

$$p = \begin{cases} \frac{T_1T_4 + T_2T_5 + T_3T_6}{T_1^2 + T_2^2 + T_3^2} = \frac{\underline{w}^T \cdot \underline{v}}{\underline{w} \cdot \underline{w}} & \text{if } \underline{w} \neq 0 \\ \infty & \text{otherwise} \end{cases}. \quad (3.1.6)$$

Definition 3 (Magnitude of a Twist). The magnitude of the twist, $\xi = \begin{pmatrix} \omega & v \end{pmatrix}^T$ is

$$\|\xi\| = \begin{cases} \|\underline{w}\| & \text{if } \underline{w} \neq 0 \\ \|\underline{v}\| & \text{otherwise} \end{cases}. \quad (3.1.7)$$

In screw theory, it is typical to concern ourselves only with the small displacements of a system's motion. Whenever a body admits an indefinitely small movement of a continuous nature, it is capable of executing that kind of movement denoted by a twist about a screw. When the body receives twists in several succession, the position that is finally attained is called the resultant twist.

Homework 2. What is the geometric meaning of (3.1.6) on a twist axis to you. Define pure rotation and a pure translation in terms of (3.1.6)¹.

¹Figure out how they correspond to zero pitch and infinite pitch twists.

3.1.2 Dynamics Screws: Twist's Pitch, Axis, and Magnitude

For a set of forces and moments applied to a rigid body, there is a wrench axis, a unique line, which has associated with it a pitch, p , and a magnitude. We may consider the set of forces and moments that act on the body to be a single force along the wrench axis and a moment about exerted about the axis. This force-moment pair is termed a wrench, and it is characterized by the 6-vector $\underline{w} = (W_1, W_2, W_3, W_4, W_5, W_6)$. We consider the first three components of \underline{w} to be the net force components, \underline{f} , exerted on the body and $\underline{m} = \{W_4, W_5, W_6\}$ to be the components of the net moment resolved at the origin of the reference frame. We define the Plücker coordinates of the wrench axis as

$$L = W_1, \quad M = W_2, \quad N = W_3 \quad (3.1.8a)$$

$$P = W_4 - pW_1, \quad Q = W_5 - pW_2, \quad R = W_6 - pW_3 \quad (3.1.8b)$$

with pitch defined as

$$p = \frac{W_1W_4 + W_2W_5 + W_3W_6}{W_1^2 + W_2^2 + W_3^2} = \frac{\underline{f} \cdot \underline{m}}{\underline{f} \cdot \underline{f}}. \quad (3.1.9)$$

Equation (3.1.9) signifies that the pitch of the wrench is the ratio of the magnitude of the applied moment about a point to the magnitude of the applied force along a wrench axis. A zero pitch wrench would therefore correspond to pure force while an infinite pitch wrench would be a pure moment. We define the magnitude of the wrench as $\|\underline{f}\| = \sqrt{W_1^2 + W_2^2 + W_3^2}$ when we have a finite pitch. If the pitch is infinite, the magnitude is $\|\underline{m}\| = \sqrt{W_4^2 + W_5^2 + W_6^2}$.

Homework 3. A unit screw, twist or wrench is one where the magnitude of the screw, twist or wrench is 1.

(1). What is the geometric meaning of a unit screw to you? (2) Consult the identified reference materials and explain what a reciprocal screw is in no more than five sentences.

3.1.3 Screw Motions

A screw motion is a rotation about the axis, l by an amount $\alpha = \ell$ followed by a translation by an amount $h\alpha$ that is parallel to the axis l . A pure translation occurs when $h = \infty$ so that the screw motion is composed of a pure translation along the axis of the screw by a distance ℓ .

3.2 Rodrigues' Formula

The matrix exponential

$$e^{\hat{\omega}t} = I + \hat{\omega}t + \frac{(\hat{\omega}t)^2}{2!} + \frac{(\hat{\omega}t)^3}{3!} + \dots \quad (3.2.1)$$

is instrumental in defining the rotation about an axis, ω :

$$R(\omega, \theta) = e^{\hat{\omega}\theta}. \quad (3.2.2)$$

Thus, for a matrix $\hat{\omega}$ in the Lie algebra $so(3)$, a unit vector $\|\omega\| = 1$, and a real number $\theta \in \mathbb{R}$, we write the exponential of $\hat{\omega}\theta$ as

$$e^{\hat{\omega}\theta} = I + \theta\hat{\omega} + \frac{\theta^2}{2!}\hat{\omega}^2 + \frac{\theta^3}{3!}\hat{\omega}^3 + \dots \quad (3.2.3)$$

Homework 4. Given a matrix $\hat{m} \in so(3)$, suppose that the following relation holds,

$$\hat{m}^2 = mm^T - \|m\|^2 I \quad (3.2.4)$$

$$\hat{m}^3 = -\|m\|^2 \hat{m} \quad (3.2.5)$$

with the fact that higher powers of \hat{m} can be recursively found. Therefore, utilizing this lemma with $m = \omega\theta$, $\|\omega\| = 1$, show that

$$e^{\hat{\omega}\theta} = I + \hat{\omega} \sin \theta + \hat{\omega}^2 (1 - \cos \theta). \quad (3.2.6)$$

Equation (3.2.6) is Rodrigues' formula.

3.3 The Matrix Exponential, The Lie Group and Lie Algebra

For any matrix

$$g = \begin{bmatrix} R & d \\ 0 & 1 \end{bmatrix} \in SE(3) \text{ such that } R \in SO(3), d \in \mathbb{R}^3, \text{ and } RR^T = I \quad (3.3.1)$$

there exists a matrix

$$N = \begin{bmatrix} S & x \\ 0 & o \end{bmatrix} \text{ such that } S = -S^T \quad (3.3.2)$$

such that $\exp(N) = g$ since we can write

$$\begin{bmatrix} R & d \\ 0 & 1 \end{bmatrix} = \begin{bmatrix} I & d \\ 0 & 1 \end{bmatrix} \begin{bmatrix} R & 0 \\ 0 & 1 \end{bmatrix} = \begin{bmatrix} I & d \\ 0 & 1 \end{bmatrix} \exp \begin{bmatrix} S & 0 \\ 0 & 0 \end{bmatrix}. \quad (3.3.3)$$

We see that the exponential map is surjective onto $\mathbb{E}(3)$. $SE(3)$ is Lie Group whose isomorphism is the lie algebra $\mathfrak{se}(3)$. Generally, we define the Lie Group $SE(n)$ on the configuration of a rigid body that consists of the pair (p_{ij}, R_{ij}) to be the product space of \mathbb{R}^n with $SO(n)$, called the special Euclidean group *i.e.*

$$SE(n) = \{(p, R) : p \in \mathbb{R}^n, R \in SO(n)\} = \mathbb{R}^n \times SO(n) \quad (3.3.4)$$

An element of $\mathfrak{se}(3)$ ² is the twist earlier introduced, which is the infinitesimal generator of the Euclidean group. Formally, we define $\xi = (\omega, v) \in \mathbb{R}^6$ as the twist coordinates of $\hat{\xi}$. Equation (3.3.3) is basically a re-statement of Euler's theorem, that is that a rigid body transformation is basically a rotation about a line passing through a preassigned fixed point. Note that S is the anti-symmetric matrix or skew-symmetric matrix with the following special

²We typically write the Lie algebra in lowercase with the math frak style.

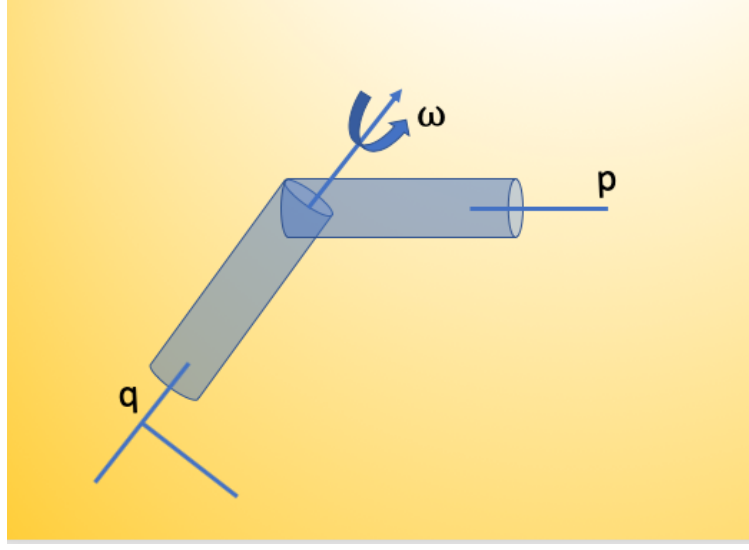


Figure 3.1: Illustration of rotation of interconnecting link of two spherical links.

property

$$S(\vec{\omega}) = \hat{\omega} = \begin{pmatrix} 0 & -\omega_z & \omega_y \\ \omega_z & 0 & -\omega_x \\ -\omega_y & \omega_x & 0 \end{pmatrix} \quad (3.3.5)$$

that is $\omega_x, \omega_y, \omega_z$ are the component of the vector $\vec{\omega}$ such that

$$s_{ij} = \begin{cases} 0, & \text{if } i = j \\ -s_{ji} & \text{if } i \neq j. \end{cases}$$

It is a common convention in robotics texts to denote the skew symmetric matrix by $\hat{\omega}$ and we will adopt that convention for the rest of these notes. In particular, we define the homogeneous coordinates for a point $q \in \mathbb{R}^3$ that is rotating about an axis $\vec{\omega} \in \mathbb{R}^3$ such that $\|\vec{\omega}\| = 1$ (see Fig. 3.1) as

$$\hat{\xi} = g^{-1}\dot{g} = \begin{pmatrix} \hat{\omega} & v \\ 0 & 0 \end{pmatrix} \in \mathfrak{se}(3) \quad (3.3.6)$$

where $v = -\omega \times q$. Equation (3.3.6) is the object's velocity in the body frame; essentially the Lie algebra element and we can obtain the twist coordinates from it via the so-called

wedge operator

$$\begin{pmatrix} \hat{\omega} \\ v \end{pmatrix}^{\wedge} = \begin{pmatrix} \hat{\omega} & v \\ 0 & 0 \end{pmatrix}. \quad (3.3.7)$$

If the link of Fig. 3.1 moves at a unit velocity, we can write the velocity at the tip point as

$$\dot{p} = \omega \times (p(t) - q(t)) \quad (3.3.8)$$

where $p(t), q(t)$ are the paths traced out by points p and q respectively during the motion.

We define the exponential for the matrix $A \in SO(n)$, which is a component of the solution to the ordinary differential equation, $\dot{x}(t) = Ax(t)$ where $x(t) \in \mathbb{R}^n$ as

The Matrix Exponential

$$e^{At} = I + At + \frac{A^2 t^2}{2!} + \frac{A^3 t^3}{3!} + \dots \quad (3.3.9)$$

For the matrix exponential of (3.3.9), we would like to write the equation in a closed-form expression since we are interested in definite solutions for our forward kinematics problems.

For suppose that some matrix $D \in \mathbb{R}^{n \times n}$ and $P \in \mathbb{R}^{n \times n}$ are available, then we find that

$$\begin{aligned} e^{At} &= I + (PDP^{-1})t + (PDP^{-1})(PDP^{-1})\frac{t^2}{2!} + \dots \\ &= P \left(IDt + \frac{(Dt)^2}{2!} + \dots \right) P^{-1} \end{aligned} \quad (3.3.10)$$

$$= Pe^{Dt}P^{-1}. \quad (3.3.11)$$

Definition 4 (Exponential Map Properties). We note the following properties of the matrix exponential:

$$(1) \quad d(e^{At}) = Ae^{At} = e^{At}A.$$

$$(2) \quad \text{For } A = PDP^{-1} \text{ for some diagonal } D \in \mathbb{R}^{n \times n} \text{ and an invertible } P \in \mathbb{R}^{n \times n}, e^{At} = Pe^{Dt}P^{-1}.$$

(3) For $AB = BA$, we have $e^A e^B = e^{A+B}$.

(4) $(e^A)^{-1} = e^{-A}$

For the mapping from the twist map in the Lie algebra to the Lie group notation, we have the following:

Exponential map from $\mathfrak{se}(3)$ to $SE(3)$

$$e^{\hat{\xi}\theta} = \begin{pmatrix} e^{\hat{\omega}\theta} & (I - e^{\hat{\omega}\theta})(w \times v) + \omega\omega^T v\theta \\ 0 & I \end{pmatrix} \in SE(3) \quad (3.3.12)$$

3.3.1 The Lie Group Notations

3.3.2 Exponential Map and Kinematic Chains

Now consider a robot manipulator of the form shown in [Fig. 2.4](#). The reader may imagine that there are right-handed triads of orthogonal vectors at the tip of each link of the chain so that the Euclidean transformation that describes the position and orientation of the $(i+1)'th$ link with the $i'th$ link is

$$\begin{pmatrix} R_i & d_i \\ 0 & 1 \end{pmatrix} \exp \begin{pmatrix} S_i & 0 \\ 0 & 0 \end{pmatrix} \theta_i = g_i \exp (\hat{\omega}_i \theta_i) \quad (3.3.13)$$

and we may imagine the triad that is fixed at the end-effector to be related to the triad at the base of the robot by the following relation

$$H(\theta_1, \theta_2, \dots, \theta_n) = R_1 e^{\theta_1 \hat{\omega}_1} R_2 e^{\theta_2 \hat{\omega}_2}, \dots, R_n e^{\theta_n \hat{\omega}_n} \quad (3.3.14)$$

and since $P \exp(M) P^{-1} = \exp(PMP^{-1})$, we can write the forward kinematic map, $g_{st} : Q \rightarrow SE(3)$ as

$$g_{st}(\theta) = e^{\theta_1 \hat{\omega}_1} e^{\theta_2 \hat{\omega}_2} \dots, e^{\theta_n \hat{\omega}_n} g_{st}(0) \quad (3.3.15)$$

using the identity repeatedly.

LIE GROUP NOTATIONS

position – orientation	standard representation $\mathbf{g} = \begin{pmatrix} \mathbf{R} & \mathbf{u} \\ 0 & 1 \end{pmatrix} \in SE(3)$	Adjoint and coAdjoint representation $Ad_{\mathbf{g}} = \begin{pmatrix} \mathbf{R} & 0 \\ \tilde{\mathbf{u}}\mathbf{R} & \mathbf{R} \end{pmatrix}, Ad_{\mathbf{g}}^* = \begin{pmatrix} \mathbf{R} & \tilde{\mathbf{u}}\mathbf{R} \\ 0 & \mathbf{R} \end{pmatrix} \in \mathbb{R}^{6 \times 6}$
velocity (body frame)	Lie Algebra element $\mathbf{g}^{-1}\dot{\mathbf{g}} = \hat{\boldsymbol{\eta}} = \begin{pmatrix} \tilde{\mathbf{w}} & \mathbf{v} \\ 0 & 0 \end{pmatrix} \in \mathfrak{se}(3)$	adjoint and coadjoint map $ad_{\boldsymbol{\eta}} = \begin{pmatrix} \tilde{\mathbf{w}} & 0 \\ \tilde{\mathbf{v}} & \tilde{\mathbf{w}} \end{pmatrix}, ad_{\boldsymbol{\eta}}^* = \begin{pmatrix} \tilde{\mathbf{w}} & \tilde{\mathbf{v}} \\ 0 & \tilde{\mathbf{w}} \end{pmatrix} \in \mathbb{R}^{6 \times 6}$ Where $\tilde{\mathbf{a}} = \begin{pmatrix} 0 & -z & y \\ z & 0 & -x \\ -y & x & 0 \end{pmatrix}$
	twist vector $\boldsymbol{\eta} = \begin{bmatrix} \mathbf{w} \\ \mathbf{v} \end{bmatrix} \in \mathbb{R}^6$	
strain (body frame)	Lie Algebra element $\mathbf{g}^{-1}\mathbf{g}' = \hat{\boldsymbol{\xi}} = \begin{pmatrix} \tilde{\mathbf{k}} & \mathbf{p} \\ 0 & 0 \end{pmatrix} \in \mathfrak{se}(3)$	adjoint and coadjoint map $ad_{\boldsymbol{\xi}} = \begin{pmatrix} \tilde{\mathbf{k}} & 0 \\ \tilde{\mathbf{p}} & \tilde{\mathbf{k}} \end{pmatrix}, ad_{\boldsymbol{\xi}}^* = \begin{pmatrix} \tilde{\mathbf{k}} & \tilde{\mathbf{p}} \\ 0 & \tilde{\mathbf{k}} \end{pmatrix} \in \mathbb{R}^{6 \times 6}$
	twist vector $\boldsymbol{\xi} = \begin{bmatrix} \mathbf{k} \\ \mathbf{p} \end{bmatrix} \in \mathbb{R}^6$	



• J. M. Selig. *Geometric Fundamentals of Robotics*. Monographs in Computer Science. Springer New York, 2007.



Figure 3.2: Lie Group and their Notations. Reprinted with permission from Federico Renda. IROS 2018, Screw Theory Tutorial.

3.4 Rigid Body Transformations

A rigid body motion is one that preserves the distance between points. In classical mechanics, we are concerned about the material description of a rigid body whereby we consider all the particles that make up the rigid body as a whole rather than treat them as a continuum as it is typical in continuum mechanics. Therefore by this logic, a rigid body is a collection of particles by which the distance between any two particles remain fixed, irrespective of the motions of the body or forces exerted on that body (Murray, 1994).

BROCKETT'S PRODUCT OF EXPONENTIALS FORMULA

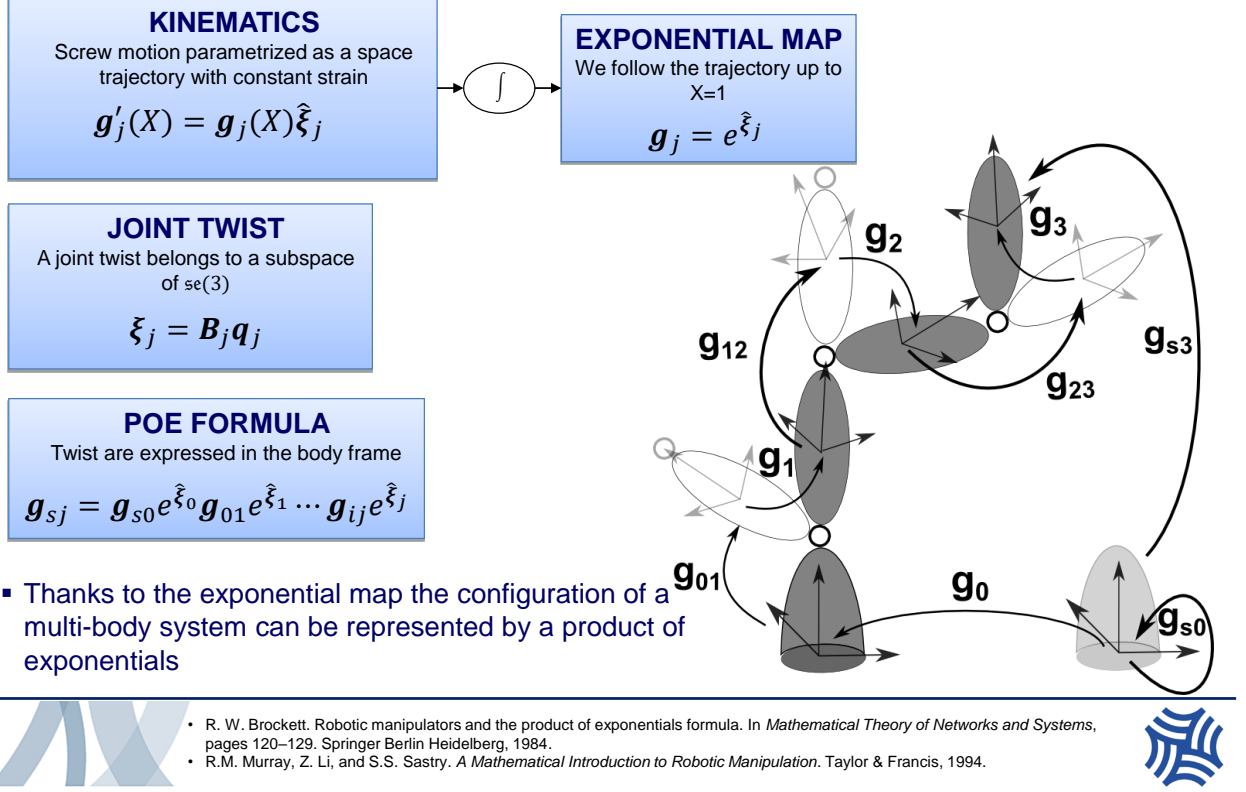


Figure 3.3: An illustration of the exponential map for a multi-body rigid system. Reprinted with permission from Federico Renda. IROS 2018, Screw Theory Tutorial.

Suppose we have two points a and b on a rigid body, we must have

$$\|a(t_f) - b(t_f)\| = \|a(t_i) - b(t_i)\| \quad (3.4.1)$$

where t_i and t_f are two instants of time that the two points *i.e.* a and b are observed on the rigid body. We thus see that distance is preserved irrespective of observer placement in rigid body transformations. For continuum-based systems such as soft robots, this is not so and we often need to come up with clever mechanisms of characterizing the motion of its particles.

Definition: A mapping $g : \mathbb{R}^3 \rightarrow \mathbb{R}^3$ is a rigid body transformation if it satisfies the following properties:

- Length is preserved *i.e.* $\|g(a) - g(b)\| = \|a - b\|$ for all point $a, b \in \mathbb{R}^3$.
- The cross product for all vectors is preserved *i.e.* $g(p \times q) = g(p) \times g(q)$ where $p, q \in \mathbb{R}^3$.

This means that the inner product is preserved so that we have,

$$p^T q = g(p)^T \times g(q). \quad (3.4.2)$$

It follows that orthogonal vectors are transformed to orthogonal vectors and since cross product is naturally preserved, rigid body transformations map orthonormal coordinate frames to orthonormal coordinate frames. Note that it is possible to have rotation of particles despite the two strong forms presented above. To track the location of a rigid body in space, it therefore follows that we need to keep track of the motion of any one point as well as the rotation of the rigid body about this point. Therefore, the configuration of the rigid body is found by attaching a Cartesian coordinate frame to a point on the rigid body and keeping track of the motion of the body coordinate frame with respect to a fixed frame. To ease representation, we typically require that all coordinate frames be right-handed: given three Cartesian orthonormal vectors $\mathbf{x}, \mathbf{y}, \mathbf{z} \in \mathbb{R}^3$ which characterize the motion of a coordinate frame, they must satisfy $\mathbf{z} = \mathbf{x} \times \mathbf{y}$.

Going by the definitions of twist in the foregoing, it follows that a twist is to a rigid body what a vector is to a point. They both express the relation needed to transfer an object from one given position to another. When a body twists at an instant, the screw about which it twists is referred to as the instantaneous screw.

3.4.1 Translation in \mathbb{R}^3

For the two coordinate frames shown in [Fig. 3.4](#), say we choose the $o_0x_0y_0$ frame as the reference and the $o_1x_1y_1$ as the moving coordinate frame, the way we would characterize the

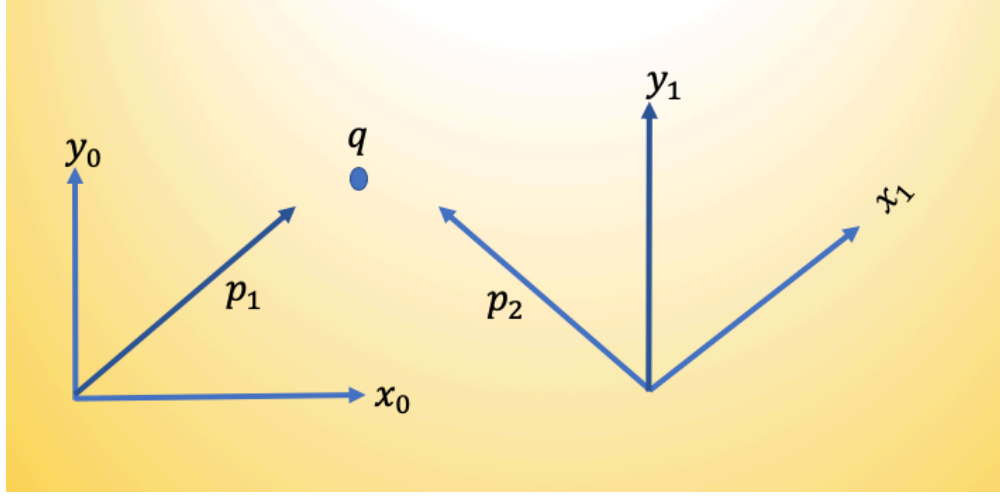


Figure 3.4: A point q in space with respect to two coordinate frames.

translation motion of the the point q would be to represent its translation from the reference frame by the Cartesian displacement along x and y so that we have

$$q^0 = \begin{pmatrix} q_x^0 \\ q_y^0 \end{pmatrix}, \quad q^1 = \begin{pmatrix} q_x^1 \\ q_y^1 \end{pmatrix} \quad (3.4.3)$$

where the superscript denotes the reference frame and the subscript denotes the coordinate of the point q along an axis in either Cartesian frame. The origin of the two frames are both points in space; therefore, the we can assign the coordinates that denote the position of the origin of one coordinate frame with respect to another as

$$o_1^0 = \begin{pmatrix} o_x^0 \\ o_y^0 \end{pmatrix}, \quad o_0^1 = \begin{pmatrix} o_x^1 \\ o_y^1 \end{pmatrix}. \quad (3.4.4)$$

For vectors such as twists or wrenches, we use a similar notation as those used for points. Thus if p_1, p_2 are two vectors that are invariant with respect to the choice of coordinate frames, we would have

$$p_1^0 = \begin{pmatrix} o_x^0 \\ o_y^0 \end{pmatrix}^T, \quad p_1^1 = R(-\theta)q^0, \quad p_2^0 = R(\theta)q^0, \quad p_2^1 = \begin{pmatrix} o_x^1 \\ o_y^1 \end{pmatrix}^T \quad (3.4.5)$$

where θ is the angle that coordinate frame 1 makes with respect to coordinate frame 0. In robotics, the standard way to apply a rotation is counterclockwise. This is the reason we negate the angle of rotation when finding the vector p_1 in frame 1. We will introduce the definition of the rotation matrix R shortly. Performing frame transformations is a fundamental step to getting a robot work as envisioned. We must ensure that all coordinate vectors are defined with respect to the same coordinate frame. We say two vectors are “equal” when they have the same magnitude and direction. Therefore, for vectors that are not constrained to be located at the same point in space, we require that they defined with respect to frames whose coordinates are parallel, given that absolute locations are not consequential, but the magnitude and direction of the vector.

3.4.2 Rotations in \mathbb{R}^3

We would like to establish a convention that all coordinate frames will be right-handed. Our goal is to establish the orientation of an object by specifying the local coordinate on the body; we then describe the body’s relative orientation between a coordinate frame attached to the body and a fixed or an inertial coordinate frame. Suppose that we have two frames I and J , where I is the inertial frame, while J is the body frame as shown in [Fig. 3.5](#). Let $\mathbf{x}_{ij}, \mathbf{y}_{ij}, \mathbf{z}_{ij} \in \mathbb{R}^3$ be the coordinates of the principal axes of J relative to I so that we have the following matrix as a result of composing the respective coordinate vectors

$$R_{ij} = \begin{bmatrix} \mathbf{x}_{ij} & \mathbf{y}_{ij} & \mathbf{z}_{ij} \end{bmatrix} = \begin{bmatrix} r_{11} & r_{12} & r_{13} \\ r_{21} & r_{22} & r_{23} \\ r_{31} & r_{32} & r_{33} \end{bmatrix}. \quad (3.4.6)$$

The resulting matrix in (3.4.6) is the rotation matrix. This matrix can be rewritten by noting that the components of a vector are the projections of that vector onto the unit directions of its reference frame. Thus, the components if R_{ij} in (3.4.6) can be written as the dot product

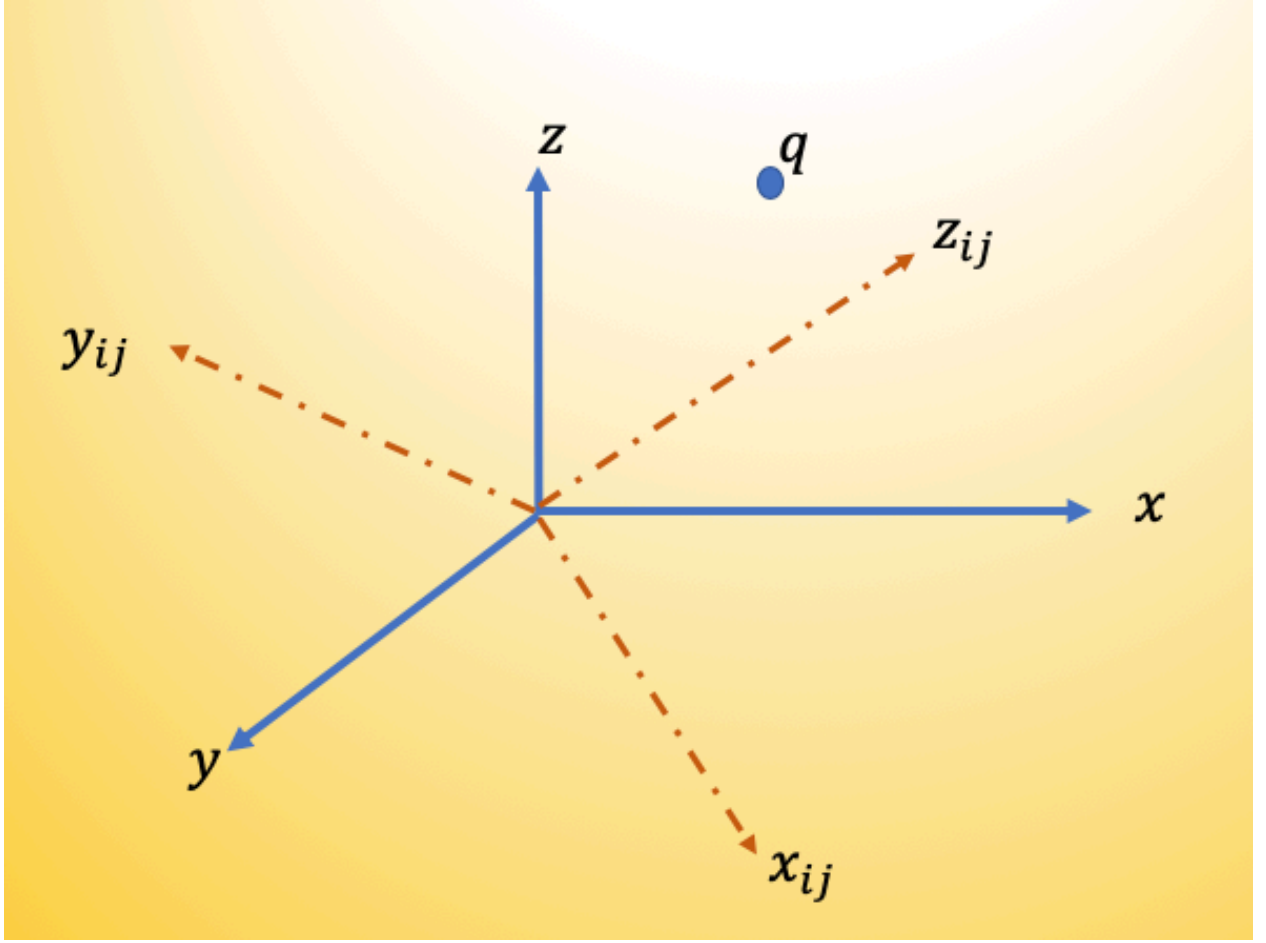


Figure 3.5: An illustration of the relative orientation of a rigid object q between an inertial frame I and a body frame J .

of a pair of unit vectors:

$$R_{ij} = \begin{bmatrix} \mathbf{x}_j \cdot \mathbf{x}_i & \mathbf{y}_j \cdot \mathbf{x}_i & \mathbf{z}_j \cdot \mathbf{x}_i \\ \mathbf{x}_j \cdot \mathbf{y}_i & \mathbf{y}_j \cdot \mathbf{y}_i & \mathbf{z}_j \cdot \mathbf{y}_i \\ \mathbf{x}_j \cdot \mathbf{z}_i & \mathbf{y}_j \cdot \mathbf{z}_i & \mathbf{z}_j \cdot \mathbf{z}_i \end{bmatrix}. \quad (3.4.7)$$

The components of the rotation matrix (3.4.6) are sometimes called direction cosines since the dot product of two unit vectors give the cosine of the angle between them as shown in (3.4.7). If we examine the rows of (3.4.7), we see that the rows of R_{ij} are the unit vectors

coordinates of I in the frame J so that we have

$$R_{ij} = R_{ji}^T \quad (3.4.8)$$

which is to say that the inverse of the rotation matrix is equal to its transpose. Noting that $\|\mathbf{r}\|^2 = x_i^2 + y_j^2 + z_i^2 = 1$, $r_i \cdot r_j = 0$ when $i \neq j$, and $r_i \cdot r_i = 1$, this can be thus verified as

$$R_{ij}^T \cdot R_{ji} = \begin{bmatrix} \mathbf{x}_j \cdot \mathbf{x}_i & \mathbf{x}_j \cdot \mathbf{y}_i & \mathbf{x}_j \cdot \mathbf{z}_i \\ \mathbf{y}_j \cdot \mathbf{x}_i & \mathbf{y}_j \cdot \mathbf{y}_i & \mathbf{y}_j \cdot \mathbf{z}_i \\ \mathbf{z}_j \cdot \mathbf{x}_i & \mathbf{z}_j \cdot \mathbf{y}_i & \mathbf{z}_j \cdot \mathbf{z}_i \end{bmatrix} \cdot \begin{bmatrix} \mathbf{x}_j \cdot \mathbf{x}_i & \mathbf{y}_j \cdot \mathbf{x}_i & \mathbf{z}_j \cdot \mathbf{x}_i \\ \mathbf{x}_j \cdot \mathbf{y}_i & \mathbf{y}_j \cdot \mathbf{y}_i & \mathbf{z}_j \cdot \mathbf{y}_i \\ \mathbf{x}_j \cdot \mathbf{z}_i & \mathbf{y}_j \cdot \mathbf{z}_i & \mathbf{z}_j \cdot \mathbf{z}_i \end{bmatrix} \quad (3.4.9)$$

$$= \begin{bmatrix} \mathbf{x}_j \cdot \mathbf{x}_j \cdot \|\mathbf{r}\|^2 & \mathbf{x}_j \cdot \mathbf{y}_j \cdot \|\mathbf{r}\|^2 & \mathbf{x}_j \cdot \mathbf{z}_j \cdot \|\mathbf{r}\|^2 \\ \mathbf{y}_j \cdot \mathbf{x}_j \cdot \|\mathbf{r}\|^2 & \mathbf{y}_j \cdot \mathbf{y}_j \cdot \|\mathbf{r}\|^2 & \mathbf{y}_j \cdot \mathbf{z}_j \cdot \|\mathbf{r}\|^2 \\ \mathbf{z}_j \cdot \mathbf{x}_j \cdot \|\mathbf{r}\|^2 & \mathbf{z}_j \cdot \mathbf{y}_j \cdot \|\mathbf{r}\|^2 & \mathbf{z}_j \cdot \mathbf{z}_j \cdot \|\mathbf{r}\|^2 \end{bmatrix} = \begin{pmatrix} 1 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 1 \end{pmatrix} \quad (3.4.10)$$

$$= \mathbf{I}_3 \quad (3.4.11)$$

where \mathbf{I}_3 is the 3×3 identity matrix.

Homework 5. Verify that $R_{ij} = R_{ji}^{-1} \equiv R_{ji}^T$. Furthermore, verify that the determinant of the rotation matrix is ± 1 i.e. $\det R = \pm 1$.

The determinant of the R matrix is written as

$$\det R = r_1^T (r_2 \times r_3). \quad (3.4.12)$$

For right-handed coordinate frames, we have that $r_2 \times r_3 = r_1$ so that $r_1^T r_1 = 1$ for a coordinate frame that aligns with the right-hand orthonormal frame representation. This special property that a 3×3 matrix satisfies $r_2 \times r_3 = r_1$ and that $\det R = r_1^T r_1 = 1$ is called the special orthogonal property, denoted $\text{SO}(3)$. Special orthogonal means $\det R = +1$. The set of all SO matrices in $\mathbb{R}^{n \times n}$ is defined by

$$\text{SO}(n) = \{R \in \mathbb{R}^{n \times n} : RR^T = \mathbf{I}, \det R = +1\}. \quad (3.4.13)$$

3.4.3 Rotation Matrices as Transformations

Suppose we are tasked with transforming a point q from one coordinate frame J to a frame I based on [Fig. 3.5](#). Suppose that $q_j = (x_j, y_j, z_j)$ are the coordinates of q with respect to the frame J . We may reason that $x_j, y_j, z_j \in \mathbb{R}^3$ are the projections of q onto the coordinate axes of B , which in turn, have coordinates $\mathbf{x}_{ij}, \mathbf{y}_{ij}, \mathbf{z}_{ij} \in \mathbb{R}^3$ with respect to coordinate frame I ; then it follows that the coordinates of q relative to frame I is

$$q_i = \mathbf{x}_{ij}x_j + \mathbf{y}_{ij}y_j + \mathbf{z}_{ij}z_j. \quad (3.4.14)$$

which in vectorized form is

$$q_i = \begin{pmatrix} \mathbf{x}_{ij} & \mathbf{y}_{ij} & \mathbf{z}_{ij} \end{pmatrix} \begin{pmatrix} x_j \\ y_j \\ z_j \end{pmatrix} = R_{ij}q_j \quad (3.4.15)$$

where the last part of the above equation follows from [\(3.4.7\)](#).

Just as a rotation matrix can act on points to transform them in the world, so can rotation matrices act on vectors. Say we have another point p_j on the frame J in [Fig. 3.5](#), then the vector that connects a point q_j in the frame J to p_j is $v_j = q_j - p_j$ so that the action of the rotation matrix on v_j is

$$R_{ij}(v_j) := R_{ij}q_j - R_{ij}p_j = q_i - p_i = v_i. \quad (3.4.16)$$

3.4.4 Planar Rotations

We now revisit the relative orientation between two coordinate frames as shown in [Fig. 3.5](#). Suppose now that the angle of rotation between the two coordinate frames is θ as shown in [Fig. 3.6](#), it follows that the composition of the rotation allows us to write

$$R_1^0 = \begin{pmatrix} x_1^0 & | & y_1^0 \end{pmatrix} \quad (3.4.17)$$

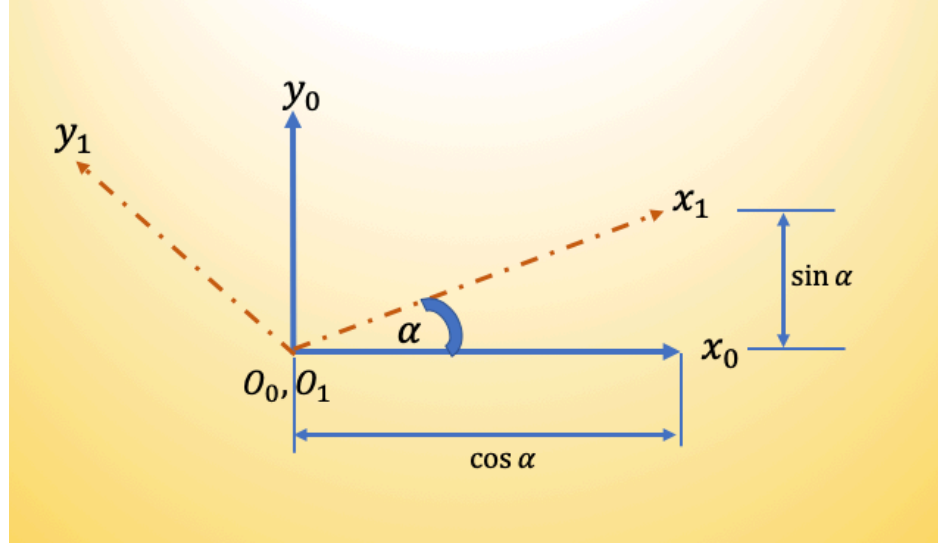


Figure 3.6: Illustration of rotation between two frames in a plane.

whereupon x_1^0 and y_1^0 have the usual meanings as before and are expressed as

$$x_1^0 = \begin{pmatrix} \cos \alpha \\ \sin \alpha \end{pmatrix} \text{ and } y_1^0 = \begin{pmatrix} -\sin \alpha \\ \cos \alpha \end{pmatrix} \quad (3.4.18)$$

so that composing the rotation matrix, we have

$$R = \begin{pmatrix} \cos \alpha & -\sin \alpha \\ \sin \alpha & \cos \alpha \end{pmatrix} \quad (3.4.19)$$

While you might find the above approach rather daunting, an easier geometric way to visualize the transformation of matrices is to recall that the columns of the rotation matrix are the direction cosines of the coordinate axes of $o_1x_1y_1$ relative to the coordinates of $o_0x_0y_0$ c.f. (3.4.7). For a planar rotation, we could extract the first 2×2 block of (3.4.7) so that

$$R_1^0 = \begin{bmatrix} \mathbf{x}_0 \cdot \mathbf{x}_1 & \mathbf{y}_1 \cdot \mathbf{x}_0 \\ \mathbf{x}_0 \cdot \mathbf{y}_1 & \mathbf{y}_1 \cdot \mathbf{y}_0 \end{bmatrix} \quad (3.4.20)$$

And since the dot product of two vectors is basically the cosine of the angles between them, we have

$$R_1^0 = \begin{bmatrix} \cos\alpha & -\cos(\pi/2 - \alpha) \\ \cos(\pi/2 - \alpha) & \cos\alpha \end{bmatrix} = \begin{bmatrix} \cos\alpha & -\sin\alpha \\ \sin\alpha & \cos\alpha \end{bmatrix} \quad (3.4.21)$$

Note the way the negative signs have entered the matrix due to the counterclockwise direction of rotation that we have chosen so as to preserve the positiveness of the determinant of R . In particular, the projection of y_1 on x_0 is negative because of our right-handed frame.

Homework 6. Compose the rotation matrix in three dimensions where all axes of the inertial frame are rotated by an angle β around each of the x_0 , y_0 and z_0 axes respectively using the foregoing logic. In addition, for each transformation, verify that (1) $R_{e,\beta} = I$ where e is the axes about which we are rotating and β is the angle of rotation, (2) the composition of rotations about the angles β and α in a successive manner implies that $R_{z,\beta}, R_{z,\alpha} = R_{z,\beta+\alpha}$, and (3) $(R_{z,\beta})^{-1} = R_{z,-\beta}$. Bonus points will be awarded for cool 3D visualizations.

Homework 7. For the two frames shown in [Fig. 3.7](#), determine the rotation matrix between them. In addition, explain the difference between rotating about a current frame and rotating about a fixed frame³. In particular, when is it necessary to carry out a pre-multiplication and when is it necessary to carry out a post-multiplication when transforming points or vectors about coordinate frames?

Order of rotations

A rotation about a fixed axis necessarily means a pre-multiplication while a rotation about a current axis necessitates a post-multiplication.

³See sections 2.4.1 and 2.4.2 of Spong's book.

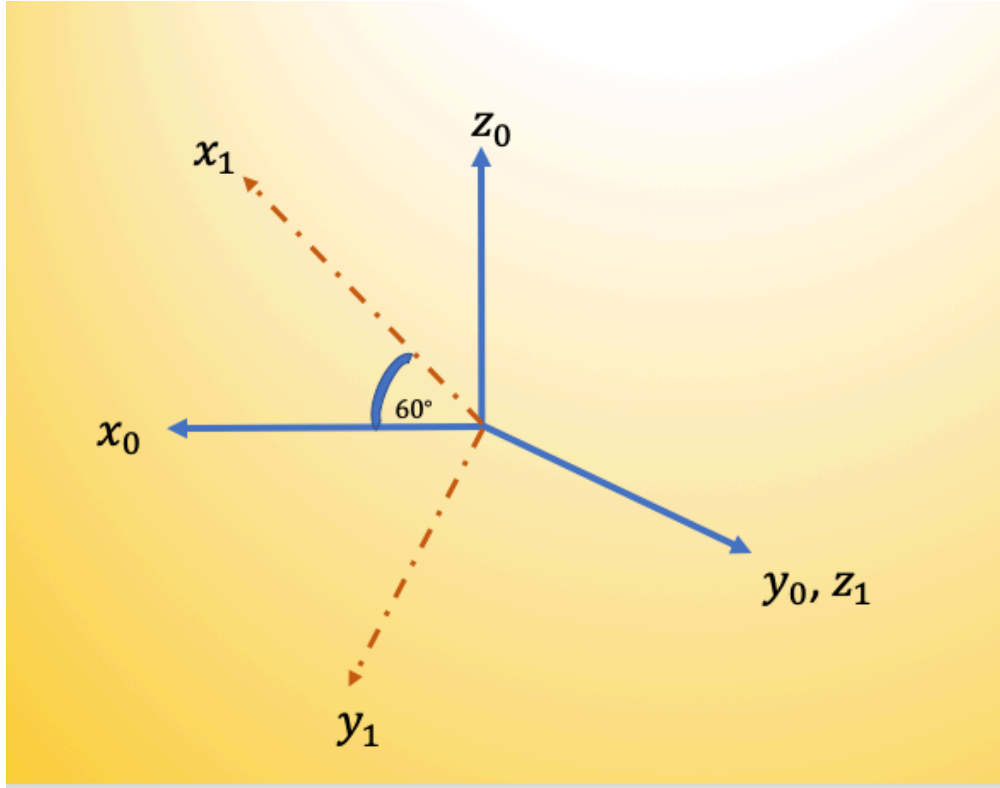


Figure 3.7: Relative orientation between two frames.

3.4.5 Composition of Rotations

Rotations matrices have this desirable property that they can be composed together to form transform a point between successive frames. Take for example a frame K whose orientation relative to frame J is R_{jk} , and frame J whose orientation relative to frame I is R_{ij} , we can write out the orientation of frame K relative to I as

$$R_{ik} = R_{ij}R_{jk}. \quad (3.4.22)$$

The rotation described above would be equivalent to rotating frame K relative to frame I according to R_{ij} , then aligning frame J to K , we rotate K relative to I according to R_{jk} . The resulting frame K has orientation with respect to I given by $R_{ij}R_{jk}$. This frame relative to which rotation occurs is termed the current frame.

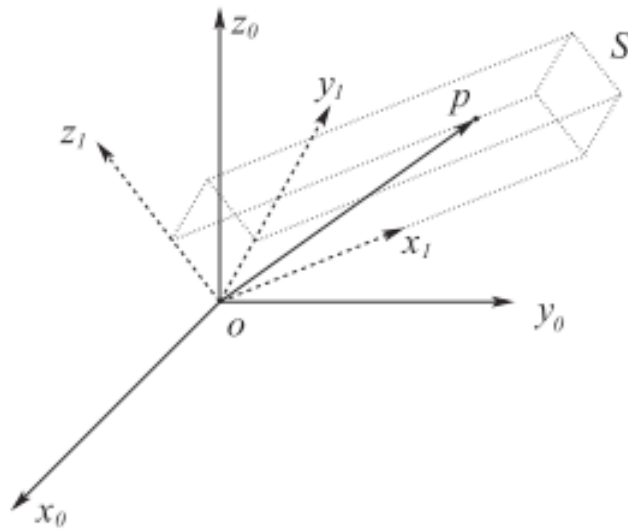


Figure 3.8: Coordinate frames on a body. Reprinted from (Spong et al., 2006)

Homework 8. For the robot manipulator we are using in this class, suppose that you have the following point in the base frame of the robot, $q_0 = [-2, 3, 1]$. Furthermore, suppose that the joint angles for all six joints are respectively $\{-90, 60, 30, 45, 90, 125\}$, transform the point q_0 in the base frame to a coordinate frame on the sixth joint.

Properties of Rigid Body Rotation Matrices

Rigid body rotations preserve

- distance: $\|Rq - Rp\| = \|q - p\|$ for all $q, p \in \mathbb{R}^3$
- orientations: $R(i \times j) = Ri \times Rj$ for all $i, j \in \mathbb{R}^3$.

What follows is adapted from Figure 2.5 from Spong and Vidyasagar's book. It is meant to illustrate the rotation applied to a point from one frame to another.

For consider Fig. 3.8 whereupon the rigid body S has a coordinate frame $o_1x_1y_1$ attached. For the coordinates of the point p with respect to frame $o_1x_1y_1$ or p^1 , we are tasked with finding the coordinates of p relative to a fixed reference frame $o_0x_0y_0$. We note that $p^1 =$

$[u, v, w]^T$ satisfies

$$p = ux_1 + vy_1 + wz_1. \quad (3.4.23)$$

Matter-of-factly, the coordinates of p^0 can be obtained based on the projection trick we introduced earlier by projecting p onto the coordinate axes of $o_0x_0y_0z_0$ so that we have

$$p^0 = \begin{pmatrix} p \cdot x_0 \\ p \cdot y_0 \\ p \cdot z_0 \end{pmatrix} \quad (3.4.24)$$

so that substituting (3.4.23) into (3.4.24), we have

$$p^0 = \begin{pmatrix} (ux_1 + vy_1 + wz_1) \cdot x_0 \\ (ux_1 + vy_1 + wz_1) \cdot y_0 \\ (ux_1 + vy_1 + wz_1) \cdot z_0 \end{pmatrix} = \begin{pmatrix} x_1 \cdot x_0 & y_1 \cdot x_0 & z_1 \cdot x_0 \\ x_1 \cdot y_0 & y_1 \cdot y_0 & z_1 \cdot y_0 \\ x_1 \cdot z_0 & y_1 \cdot z_0 & z_1 \cdot z_0 \end{pmatrix} \begin{pmatrix} u \\ v \\ w \end{pmatrix} \quad (3.4.25)$$

which upon inspection turns out to be a multiplication of the rotation matrix that transforms point p^1 to point p^0 i.e.

$$p^0 = R_1^0 p^1, \quad (3.4.26)$$

implying that the rotation matrix not only serves to represent the relative orientation of coordinate frames with respect to one another but to transform coordinates of a point from one frame to another.

Finally, we define a similarity transformation as the matrix representation of a general linear transformation that is transformed from one frame to another. So if I is the matrix representation of a linear transformation in a frame $o_0x_0y_0z_0$ and J is the equivalent transformation in $o_1x_1y_1z_1$ then I and J are related by

$$J = (R_1^0)^{-1} A R_1^0 \quad (3.4.27)$$

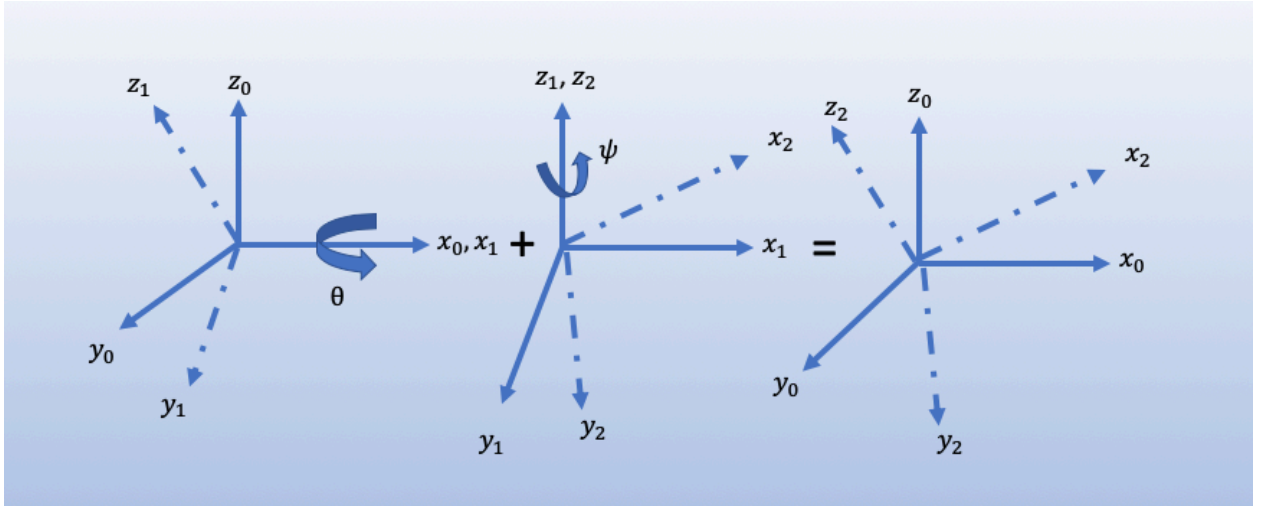


Figure 3.9: Illustration of composition of rotations about a current axis.

where R_1^0 is the coordinate transformation between frames $o_1x_1y_1z_1$ and $o_0x_0y_0z_0$.

Lab Exercise: Carry out a similarity transformation in the elbow frame ($o_1x_1y_1z_1$) of the manipulator with respect to the shoulder frame ($o_0x_0y_0z_0$) when the two frames are related by the rotation

$$R = \begin{pmatrix} 1 & 0 & 0 \\ 0 & -1 & 0 \\ 0 & 0 & 0 \end{pmatrix} \quad (3.4.28)$$

Consider the composition of rotations of Fig. 3.9 where we first rotate by an angle θ about the x axis and then rotate about an angle ψ about the z axis. The rotation matrix can be composed as

$$R = R_{x,\theta}R_{z,\psi} = \begin{pmatrix} 1 & 0 & 0 \\ 0 & c_\theta & -s_\theta \\ 0 & s_\theta & c_\theta \end{pmatrix} \cdot \begin{pmatrix} c_\psi & -s_\psi & 0 \\ s_\psi & c_\psi & 0 \\ 0 & 0 & 1 \end{pmatrix} = \begin{pmatrix} c_\psi & -s_\psi & 0 \\ 0 & c_\theta c_\psi & -s_\theta \\ s_\theta s_\psi & s_\theta c_\psi & c_\theta \end{pmatrix} \quad (3.4.29)$$

Notice how the order of multiplication is carried out, owing to the axis about which we are making the transformation.

Homework 9. Carry out the transformation above in reverse order. What do you notice?

3.5 Parameterization of Rotations $\in SO(3)$

3.5.1 Axis-Angle Parameterizations

The exponential coordinates are the canonical coordinates of the rotation group. A rigid body has at most three rotational degrees of freedom so that we need at most three variables to denote its orientation in the world. For the matrix

$$R = \begin{bmatrix} r_{11} & r_{12} & r_{13} \\ r_{21} & r_{22} & r_{23} \\ r_{31} & r_{32} & r_{33} \end{bmatrix} \quad (3.5.1)$$

so that equating the foregoing with the exponential map for a rotation about θ around an axis ω as yields

$$e^{\hat{\omega}\theta} = I + \hat{\omega} \sin \theta + \hat{\omega}^2 (1 - \cos \theta) \quad (3.5.2)$$

$$= \begin{bmatrix} \omega_y^2 v_\theta + c_\theta & \omega_x \omega_y v_\theta - \omega_z s_\theta & \omega_x \omega_z v_\theta + \omega_y s_\theta \\ \omega_x \omega_y v_\theta + \omega_z s_\theta & \omega_y^2 v_\theta + c_\theta & \omega_y \omega_z v_\theta - \omega_x s_\theta \\ \omega_x \omega_z v_\theta - \omega_y s_\theta & \omega_y \omega_z v_\theta + \omega_x s_\theta & \omega_z^2 v_\theta + c_\theta \end{bmatrix}, \quad (3.5.3)$$

where we have used $v_\theta = 1 - \cos \theta$. Therefore, we have

$$\text{trace}(R) = r_{11} + r_{22} + r_{33} = 1 + 2c_\theta \quad (3.5.4)$$

Inspecting the matrix of (3.5.3), we have the angle of rotation in terms of the three components of the rotation matrix as

$$\theta = \cos^{-1} \left(\frac{\text{trace}(R) - 1}{2} \right) \quad (3.5.5)$$

where θ is defined under the constraint, $-2\pi n \leq \theta \leq 2\pi n$ owing to the special property of R .⁴

⁴Since R preserves lengths and $\det R = +1$, the eigenvalues of R have a unit magnitude and occur in complex conjugate pairs. This implies $-1 \leq \text{trace}(R) \leq 3$

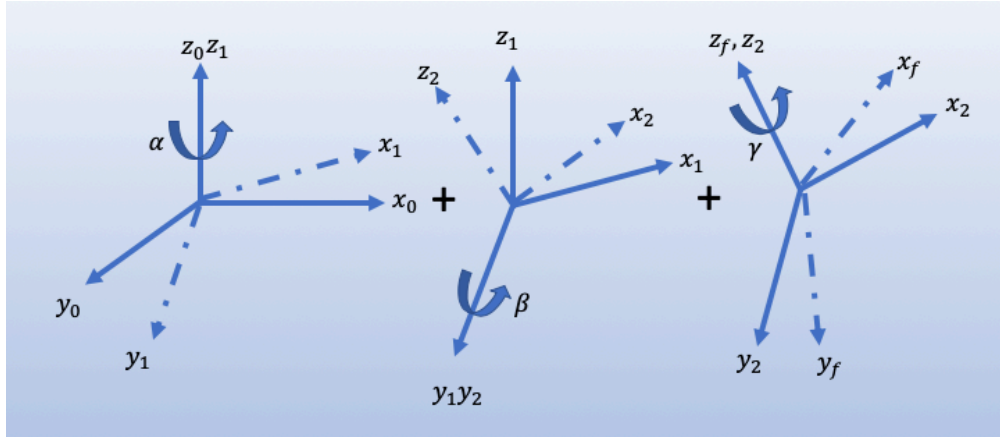


Figure 3.10: An illustration of the ZYZ rotation angles

Homework 10. Verify by equating the off-diagonal terms that

$$\omega = \frac{1}{2 \sin \theta} \begin{pmatrix} r_{32} - r_{23} \\ r_{13} - r_{31} \\ r_{21} - r_{12} \end{pmatrix} \quad (3.5.6)$$

suppose $\theta \neq 0$. Equation (3.5.5) together with (3.5.6) are what we call the axis-angle representation.

3.5.2 Euler Angles

The Euler Angles are particularly useful for specifying the orientation of a coordinate frame J with respect to I . To do this, we start as follows (see Fig. 3.10 for reference):

- we orientate frame J with frame I so that the two frames are coincident by rotating by the z axis about an angle α ;
- we then rotate about the new y -axis of J by an angle β ;
- lastly, we rotate about the new z -axis of rge frame J by an angle γ

so that altogether, we have a rotation $R_{ij}(\alpha, \beta, \gamma)$ given as

$$R_{ij}(\alpha, \beta, \gamma) = R_z(\alpha)R_y(\beta)R_z(\gamma) \quad (3.5.7)$$

$$= \begin{bmatrix} c_\alpha & -s_\alpha & 0 \\ s_\alpha & c_\alpha & 0 \\ 0 & 0 & 1 \end{bmatrix} \begin{bmatrix} c_\theta & 0 & s_\theta \\ 0 & 1 & 0 \\ -s_\theta & 0 & c_\theta \end{bmatrix} \quad (3.5.8)$$

$$= \begin{bmatrix} c_\alpha c_\beta c_\gamma - s_\alpha s_\gamma & -c_\alpha c_\beta s_\gamma - s_\alpha c_\gamma & c_\alpha s_\beta \\ s_\alpha c_\beta c_\gamma + c_\alpha s_\gamma & -s_\alpha c_\beta s_\gamma + c_\alpha c_\gamma & s_\alpha s_\beta \\ -s_\beta c_\gamma & s_\beta s_\gamma & c_\beta \end{bmatrix}$$

where as before the abbreviations c_α, s_α are abbreviations for $\cos \alpha$ and $\sin \alpha$. Euler angles arise when we need to recover the α, β, γ angles from (3.5.8). Suppose that r_{13} and r_{23} are both $\neq 0$, we find that $c_\beta \neq \pm 1$ and thus with the knowledge that $\sin \beta > 0$ given (3.5.9a), we find that ,

$$\beta = \arctan 2(r_{33}, \sqrt{1 - r_{33}^2}) \quad (3.5.9a)$$

$$\alpha = \arctan 2(r_{23}/\sin \beta, r_{13}/\sin \beta) \quad (3.5.9b)$$

$$\gamma = \arctan 2(r_{32}/\sin \beta, -r_{31}/\sin \beta) \quad (3.5.9c)$$

where $\arctan 2(y, x)$ computes $\tan^{-1}(y/x)$, determining the quadrant of the angle based on the sign of x and y . When $\sin \beta < 0$, we find that

$$\beta = \arctan 2(r_{33}, -\sqrt{1 - r_{33}^2}) \quad (3.5.10a)$$

$$\alpha = \arctan 2(-r_{23}/\sin \beta, -r_{13}/\sin \beta) \quad (3.5.10b)$$

$$\gamma = \arctan 2(-r_{32}/\sin \beta, r_{31}/\sin \beta) \quad (3.5.10c)$$

implying that the Euler angles are not unique owing to the sign of the angle about which the y axis rotates.

Homework 11. What happens when $r_{13} = r_{23} = 0$? Can you write out the rotation matrix as well as the euler angles for this situation?

In the scenario that results from homework viii, there will be infinitely many solution owing to the fact that only $\alpha + \beta$ only can be determined. This infinite solutions occur wehn $R = I$ and an example scenario of when this occurs is when $(\alpha, 0, -\alpha)$.

Using the ZYZ angles are not the only way of parameterizing the rotation matrix. We could permute the order of rotation or rotate successively about different axis. Examples include ZYX axes rotations (or Fick angles) and the YZX axes parameterization or Helmholtz angles. In general robotics speak, note that the ZYX angles are otherwise referred to as the yaw, pitch and roll angles, wherein the rotation matrix R_{ij} is defined by rotating about the x -axis in the body frame (roll), then the y -axis in the body frame (pitch), and finally the z -axis in the body frame (yaw). An advantage of the Fick angle and Helmholtz angle parameterization is that they avoid singularity at $R = I$ though they do contain singularities at other configurations.

3.5.3 Quaternions

Rather than use rotation matrices to represent orientations, a more effective approach of representing orientations are quaternions. Intead of locally parameterizing the $SO(3)$ Lie group, quaternions, unlike rotation matrices, globally parameterize the $SO(3)$ Lie Group. Formally, we represent a quaternion as follows:

$$Q = q_0 + q_x \mathbf{i} + q_y \mathbf{j} + q_z \mathbf{k}, \quad q_{i \in \{0, \dots, 3\}} \in \mathbb{R} \quad (3.5.11)$$

where q_0 is the scalar component of Q and $\mathbf{q} = (q_x, q_y, q_z)$ is the vector component.

The unit quaternions are the subset of all $Q \in \mathbb{Q}$ such that $\|Q\| = 1$; for a rotation matrix $R = \exp(\hat{\omega}\theta)$, we have the unit quaternion as

$$Q = (\cos(\theta/2), \omega \sin(\theta/2)), \quad (3.5.12)$$

where $\omega \in \mathbb{R}^3$ is the axis of orientation and $\theta \in \mathbb{R}$ is the angle of rotation.

Summary of Parameterizations

Rotation matrices can be parameterized in one of many ways depending on our use case. The common examples of parameterizations are

- (1) Axis-Angle representation;
- (2) Euler angles (ZYZ) representation;
- (3) Fick angles (*i.e.* ZYX or yaw, pitch and roll) representation;
- (4) Helmholtz angles (or YZX) angles representation; and
- (5) Quaternions.

3.6 Homogeneous Coordinates

For a point $q \in \mathbb{R}^3$, we represent the homogeneous coordinates of point q as

$$\bar{q} = \begin{pmatrix} q_x & q_y & q_z & 1 \end{pmatrix}^T \quad (3.6.1)$$

whose origin has the following coordinates

$$\bar{O} = \begin{pmatrix} 0 & 0 & 0 & 1 \end{pmatrix}^T. \quad (3.6.2)$$

For vectors, it would suffice for us to write

$$\bar{v} = \begin{pmatrix} v_x & v_y & v_z & 0 \end{pmatrix}^T \quad (3.6.3)$$

where the last element is zero because a vector is the difference of two points.

Vectors and Points

- The sum or difference of two vectors results in a vector
- The difference between two points is a vector
- The sum of two points do not exist

We now define the homogeneous transformation of a point q_j in frame J with respect to a coordinate frame I with p_{ij} being the distance between q_i , and q_j , and R_{ij} being the rotation matrix that transforms points in frame J to points in frame I . We write

$$q_i = p_{ij} + R_{ij}q_j \quad (3.6.4)$$

or more appropriately

$$\begin{pmatrix} q_i \\ 1 \end{pmatrix} = \begin{pmatrix} R_{ij} & p_{ij} \\ 0 & 1 \end{pmatrix} \begin{pmatrix} q_j \\ 1 \end{pmatrix} \quad (3.6.5)$$

which implies that $\bar{q} = \bar{g}_{ij}\bar{q}_j$. We say \bar{g}_{ij} is the homogeneous representation of $g_{ij} \in SE(3)$.

Similar to rotation matrices, rigid body transformation matrices can be composed to form new rigid body transformations. So, suppose we have the g_{jk} which is the transformation of a body K relative to body J and g_{ij} which is the transformation of body J relative to body I , then we can find the configuration of K relative to I as follows

$$\bar{g}_{ik} = \bar{g}_{ij}\bar{g}_{jk} = \begin{pmatrix} R_{ij} & p_{ij} \\ 0 & 1 \end{pmatrix} \begin{pmatrix} R_{jk} & p_{jk} \\ 0 & 1 \end{pmatrix} \quad (3.6.6)$$

$$= \begin{pmatrix} R_{ij}R_{jk} & R_{ij}p_{jk} + p_{ij} \\ 0 & 1 \end{pmatrix}. \quad (3.6.7)$$

Since (3.6.7) is a form of (3.3.4), we conclude that the composition of rigid body transformations and $I_{4 \times 4}$ is in the special Euclidean group as well. In addition,

$$g^{-1} = (R^T p, R^T). \quad (3.6.8)$$

3.7 Rigid Body Velocities

We are concerned with the velocity of a rigid body with motion described by a time-parameterized curve $g(t) \in SE(3)$. For a start, we will consider the trajectory motion of an object frame J whose origin is at a frame I and it is rotating relative to the fixed frame I . We call frame I the spatial frame and the frame J , the body frame.

3.7.1 Rotational Velocities

For a point q whose coordinates q_j are fixed in the body frame, a path in spatial coordinates is followed given by

$$q_i(t) = R_{ij}(t)q_j. \quad (3.7.1)$$

The velocity in spatial coordinates is

$$v_{q_i}(t) = \frac{\partial}{\partial t} q_i(t) = \dot{R}_{ij}(t)q_j. \quad (3.7.2)$$

We see that \dot{R}_{ij} maps the body coordinates, q_j to the spatial velocity of q . We would like to develop a more compact representation of the orientation of the point q relative to the spatial frame I by exploiting the special structure of \dot{R}_{ij} . Thus, we write

$$v_{q_i}(t) = \dot{R}_{ij}(t)R_{ij}^{-1}(t)R_{ij}(t)q_j. \quad (3.7.3)$$

We define the instantaneous spatial angular velocity $\hat{\omega}_{ij}^s \in \mathbb{R}^3$ as seen in the spatial frame I as

$$\hat{\omega}_{ij}^s(t) = \dot{R}_{ij}(t)R_{ij}^{-1}(t) \quad (3.7.4)$$

and we define the instantaneous body angular velocity as seen in the body frame J as

$$\hat{\omega}_{ij}^b(t) = R_{ij}^{-1}(t)\dot{R}_{ij}(t) \quad (3.7.5)$$

We define the body angular velocity as viewed from the instantaneous body frame J as

$$\hat{\omega}_{q_b}^b = R_{ij}^{-1}\hat{\omega}_{ij}^s R_{ij}, \text{ or } \hat{\omega}_{ij}^b = R_{ij}^{-1}\hat{\omega}_{ij}^s, \quad (3.7.6)$$

so that the body angular velocity can be recovered from the spatial angular velocity by rotating the angular velocity vector into the instantaneous body frame. Thus, (3.7.3) becomes

$$v_{q_i}(t) = \hat{\omega}_{ij}^s(t)R_{ij}(t)q_j = \hat{\omega}_{ij}^s(t) \times q_i(t) \quad (3.7.7)$$

Similarly, the velocity in the frame J can be derived from (3.7.1) as

$$q_j = R_{ij}^{-1}(t)q_i(t) \quad (3.7.8a)$$

$$v_{q_j}(t) = \dot{R}_{ij}^T(t) v_{q_i}(t) = \omega_{ij}^b(t) \times q_j. \quad (3.7.8b)$$

Thus we have the compact description of the rigid body particles' velocities in both the body and spatial angular frames as ω_{ij}^b and ω_{ij}^s .

Homework 12. Consider the motion of a point body about the x axis of an orthogonal triad with respect to a certain spatial frame located at $O(0,0,0)$. Determine the body and spatial velocities of the point with respect to the origin, O .

3.7.2 Rigid Body Velocity

We shall denote the rigid body motion of a frame J attached to a body so that is it fixed relative to a frame I by $g_{ij}(t)$. We write the velocity in the spatial frame as $\dot{g}_{ij}g_{ij}^{-1}$ which is symbolically $(\dot{p}_{ij}, \dot{R}_{ij}) \cdot (R_{ij}^T, -R_{ij}^T p_{ij})$ following the notation we introduced in (3.6.8). This has an isomorphism of a twist and it is given by

Twist in Spatial Frame

$$\eta_{ij}^s = \dot{g}_{ij}g_{ij}^{-1} = \begin{bmatrix} \omega_{ij}^s & v_{ij}^s \\ 0 & 0 \end{bmatrix} = \begin{bmatrix} \dot{R}_{ij}R_{ij}^T & -\dot{R}_{ij}R_{ij}^T p_{ij} + \dot{p}_{ij} \\ 0 & 0 \end{bmatrix} \in \mathfrak{se}(3) \quad (3.7.9)$$

whereupon the twist coordinates $\xi^s \in \mathbb{R}^6$ in the body frame can be recovered as

$$\begin{pmatrix} v_{ij}^s \\ \omega_{ij}^s \end{pmatrix} = \begin{pmatrix} -\dot{R}_{ij}R_{ij}^T p_{ij} + \dot{p}_{ij} \\ (\dot{R}_{ij}R_{ij}^T)^\vee \end{pmatrix} \quad (3.7.10)$$

Similarly, in body coordinates, we find that the twist is given by

Twist in Body Frame

$$\eta_{ij}^b = g_{ij}^{-1} \dot{g}_{ij} = \begin{bmatrix} \omega_{ij}^b & v_{ij}^b \\ 0 & 0 \end{bmatrix} = \begin{bmatrix} \dot{R}_{ij} R_{ij}^T & R_{ij}^T p_{ij} \dot{p}_{ij} \\ 0 & 0 \end{bmatrix} \in \mathfrak{se}(3) \quad (3.7.11)$$

whereupon the twist coordinates $\xi^b \in \mathbb{R}^6$ in the body frame can be recovered as

$$\begin{pmatrix} v_{ij}^b \\ \omega_{ij}^b \end{pmatrix} = \begin{pmatrix} -\dot{R}_{ij}^T \dot{p}_{ij} \\ (R_{ij}^T \dot{R}_{ij})^\vee \end{pmatrix} \quad (3.7.12)$$

Furthermore, we have that

$$\eta_{ij}^s = \begin{pmatrix} -\dot{R}_{ij} & \hat{p}_{ij} R_{ij} \\ 0 & R_{ij} \end{pmatrix} \begin{pmatrix} v_{ij}^b \\ \omega_{ij}^b \end{pmatrix}. \quad (3.7.13)$$

Homework 13. Confirm equations (3.7.9), (3.7.11) and (3.7.13).

The matrix which transforms from the body coordinate frame to the spatial velocity frame is the so-called adjoint transformation of g (c.f. Fig. 3.2), defined as

$$Ad_g = \begin{pmatrix} R & \hat{p}R \\ 0 & R \end{pmatrix} \quad (3.7.14)$$

and whose inverse is given by

$$Ad_g^{-1} = \begin{pmatrix} R^T & -(R^T p)^\wedge R^T \\ 0 & R^T \end{pmatrix} = \begin{pmatrix} R^T & -R^T \hat{p} \\ 0 & R^T \end{pmatrix} = Ad_{g^{-1}} \quad (3.7.15)$$

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