## **Accuracy of original MPM**

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#### **Outline**

- "Original" MPM
- Numerical accuracy
- Benchmarks
  - Vibrating bar
  - Oedometer
- Accuracy of FEM
- Grid crossing



## "Original" MPM

- Modified Lagrangian algorithm
- Euler-Cromer time integration
- Piecewise linear basis functions
- No additional interventions



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- Modified Lagrangian algorithm
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- Own MATLAB implementation



## **Numerical accuracy**

#### **Numerical Approximation**

$$u_{ex} = u_{num} + O(\Delta x^n) + O(\Delta t)$$

#### **RMS Error**

$$Error_{RMS} = \sqrt{\frac{1}{n_p} \left( \sum_{p=1}^{n_p} u_{num}(x_p, t) - u_{ex}(x_p, t) \right)^2}$$

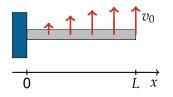
#### Accuracy in displacement

For  $\Delta t \to 0$ , the order of accuracy is equal to n, i.e. the reduction of  $\Delta x$  by a factor of 2 decreases the RMS error by  $2^n$ .





### Vibrating bar



$$\frac{\partial^2 u}{\partial t^2} = \frac{E}{\rho} \frac{\partial^2 u}{\partial x^2}$$

#### Boundary conditions:

$$u(0,t)=0$$

$$\frac{\partial u}{\partial x}(L,t)=0$$

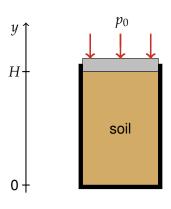
Initial conditions:

$$u(x,0)=0$$

$$\frac{\partial u}{\partial t}(x,0) = v_0 \sin\left(\frac{\pi x}{2L}\right)$$



#### **Oedometer**



$$\frac{\partial^2 u}{\partial t^2} = \frac{E}{\rho} \frac{\partial^2 u}{\partial^2 y} - g$$

Boundary conditions:

$$u(0,t)=0$$

$$\frac{\partial u}{\partial y}(H,t) = \frac{p_0}{E}$$

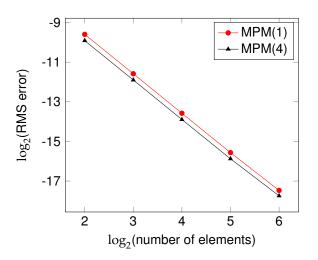
Initial conditions:

$$u(y,0)=0$$

$$\frac{\partial u}{\partial t}(y,0) = 0$$

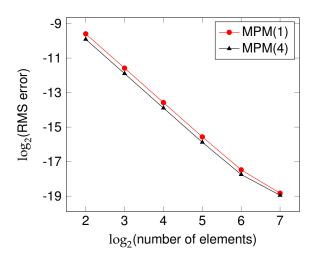


## Accuracy: vibrating bar





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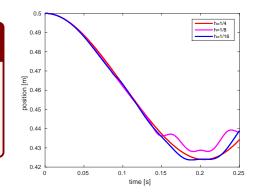


## **Accuracy: oedometer**

# Richardson's extrapolation

The order of accuracy *n* is obtained from

$$\frac{u_{num}(2h) - u_{num}(4h)}{u_{num}(h) - u_{num}(2h)} = 2^{n}.$$

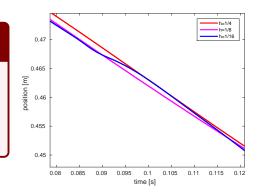


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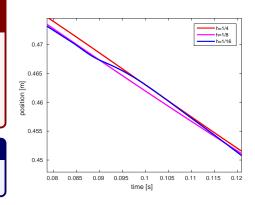
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#### Conclusion

Lack of convergence.



#### **FEM:** oedometer

#### Theoretical order of accuracy

k + 1, where k is the order of the interpolating polynomials.



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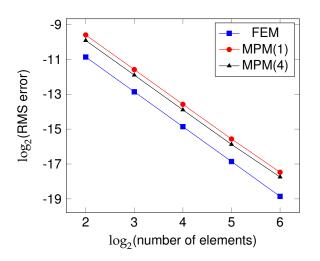
#### **Observations**

- Lack of convergence
- Problems arise due to external forces

$$\mathbf{M} \frac{d\mathbf{v}}{dt} = \mathbf{F}_{ext} - \mathbf{F}_{int},$$
  
where  $\mathbf{F}_{ext} = \mathbf{N}(H)^T p_0 - \int_0^H \mathbf{N}^T \rho g dy$ 

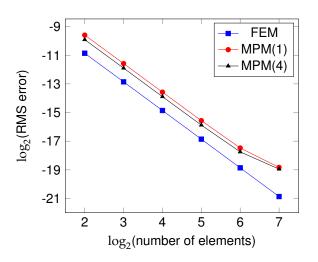


### FEM: vibrating bar



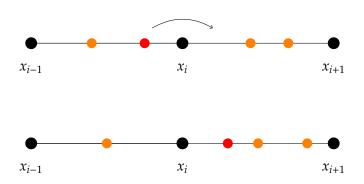


### FEM: vibrating bar



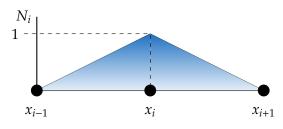


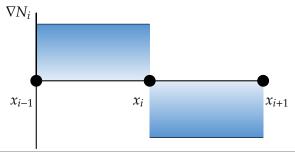
## **Grid-crossing**





## Grid-crossing: properties of shape functions





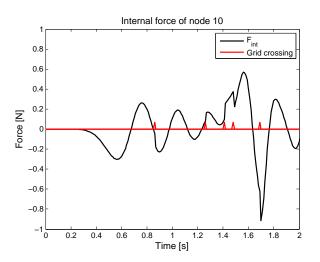


## Grid crossing: internal force

$$\begin{split} F_{i+1}^{int} &\approx \sum_{p=1}^{n_i} \nabla N_i(\xi_p) \sigma_p \Omega_p + \sum_{p=1}^{n_{i+1}} \nabla N_i(\xi_p) \sigma_p \Omega_p \\ F_{i+1}^{int} &\approx \sigma \Omega(n_i - n_{i+1}) \\ \begin{cases} F_{i+1}^{int} &= 0, & \text{if } n_i = n_{i+1} \\ F_{i+1}^{int} &\neq 0, & \text{otherwise} \end{cases} \end{split}$$



## Grid crossing: internal force





## Grid crossing: vibrating bar

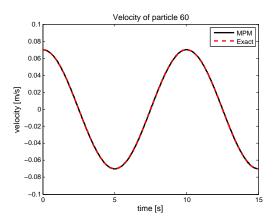


Figure: No grid crossing (30 elements).



## Grid crossing: vibrating bar

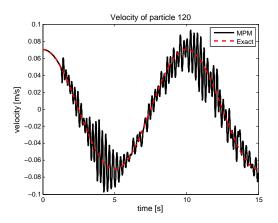
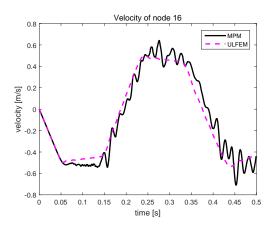


Figure: Grid crossing (60 elements).



## **Oedometer: velocity**







## Settings: vibrating bar

	Symbol	Value	Unit
Length	L	25	m
Tension	Ε	100	Pa
Density	ρ	1	kg/m <sup>3</sup>
Maximum velocity	$v_0$	0.1	m/s
Time step	$\Delta t$	$1\cdot 10^{-3}$	S
Measurement time <sup>1</sup>	t	0.5	S
$PPC^2$		4	



## **Settings: oedometer**

	Symbol	Value	Unit
Height	L	1	m
Young's modulus	Ε	$1 \cdot 10^5$	Pa
Density	ρ	$1 \cdot 10^3$	kg/m <sup>3</sup>
Load	$p_0$	0	Pa
Gravitational acceleration	g	9.81	m/s <sup>2</sup>
Time step	$\Delta t$	$1\cdot 10^{-3}$	S
Measurement time <sup>1</sup>	t	0.5	S
Position particle <sup>1</sup>	$x_p$	$\approx 0.5$	m
Number of elements <sup>2</sup>	,	30	
$PPC^2$		10	



## **Settings: Steffen**

	Symbol	Value	Unit
Length	L	1	m
Tension	Ε	100	Pa
Density	ρ	100	kg/m³
Load	$p_0$	0.7	Pa
Gravitational acceleration	8	0	$m/s^2$
Time step	$\Delta t$	$1 \cdot 10^{-2}$	S
Domain		1.15	m
Number of elements		20	
PPC		10	

