

# Master's Thesis

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## Abstract

Fill it.

## 1 Introduction

## 2 Motivation

## 3 Method

The idea of asymptotic homogenization. In a repeating cell  $Y$ ,

$$\sigma_{ij} = C_{ijkl} \epsilon_{kl} \quad (1)$$

where  $C_{ijkl}(\underline{x} + \underline{y}Y) = C_{ijkl}(\underline{x})$

$$\Rightarrow C_{ijkl}(x_1 + n_1 Y_1 \ x_2 + n_2 Y_2 \ x_3 + n_3 Y_3) = C_{ijkl}(x_1, x_2, x_3) \quad (2)$$

$C_{ijkl}(\underline{x})$  is  $Y$ -periodic

$$\underline{y} = \frac{\underline{x}}{\epsilon} \quad (3)$$

$$\Rightarrow g = g(\underline{x}, \frac{\underline{x}}{\epsilon}) = g(\underline{x}, \underline{y}) \quad (4)$$

$\underline{x} = (x_1, x_2, x_3) \in \mathbb{R}^3$  defines the domain of the composite  $\Omega$ . The domain is composed of base cells of dimensions,  $\epsilon Y_1, \epsilon Y_2, \epsilon Y_3$  where  $\underline{y} = \frac{\underline{x}}{\epsilon}$

### 3.1 1D Elasticity

$$\sigma^\epsilon = E^\epsilon \frac{\partial u^\epsilon}{\partial x} \quad (5)$$

$$\frac{\partial \sigma^\epsilon}{\partial x} + \gamma^\epsilon = 0 \quad E^\epsilon \gamma^\epsilon \rightarrow \text{macroscopically uniform} \quad (6)$$

Inside each cell,

$$E^\epsilon(x, \frac{x}{\epsilon}) = E(y) \quad (7)$$

$$\gamma^\epsilon(x, \frac{x}{\epsilon}) = \gamma(y) \quad (8)$$

Let

$$u^\varepsilon(x) = u^0(x, y) + \varepsilon u^1(x, y) + \varepsilon^2 u^2(x, y) + \dots \quad (9)$$

$$\sigma^\varepsilon(x) = \sigma^0(x, y) + \varepsilon \sigma^1(x, y) + \varepsilon^2 \sigma^2(x, y) + \dots \quad (10)$$

### 3.2 Optimal Design of Elastic structures

$\mathbf{b} \rightarrow$  body forces

$\mathbf{t} \rightarrow$  surface tractions

Optimal choice of  $\mathbb{C}_{ijkl} \in U_{ad} \leftarrow$  admissible set of elasticity ??

$\mathbb{C}_{ijkl}(\mathbf{x}) \forall \mathbf{x} \in \Omega$  has 21 independent components

$a_E(\mathbf{u}, \mathbf{v}) = \int_{\Omega} \mathbb{C}_{ijkl} \varepsilon_{kl}(\mathbf{u}) \varepsilon_{kl}(\mathbf{v}) d\mathbf{v} \rightarrow$  energy bilinear form

$L(\mathbf{v}) = \int_{\Omega} \mathbf{v} d\mathbf{x} + \int_{\partial\Omega_t} \mathbf{t} \cdot \mathbf{v} ds \rightarrow$  load linear form.

Minimum compliance problem:

$$\text{minimize} \quad L(\mathbf{v}), \quad (11)$$

$$\text{subject to} \quad \mathbb{C}_{ijkl} \in \mathbb{U}_{ad} \quad (12)$$

$$a_E(\mathbf{u}, \mathbf{v}) = L(\mathbf{v}) \quad \forall \mathbf{v} \in \mathbb{U} \quad (13)$$

where  $\mathbb{U} \rightarrow$  kinematically admissible displacements.

For optimal shape design:

$$\mathbb{C}_{ijkl}(\mathbf{x}) = \chi(\mathbf{x}) \bar{\mathbb{C}}_{ijkl}, \quad \text{where } \bar{\mathbb{C}}_{ijkl} \rightarrow \text{stiffness matrix of the material} \quad (14)$$

$$\chi(\mathbf{x}) = \begin{cases} 1 & \text{if } \mathbf{x} \in \Omega^m, \\ 0 & \text{if } \mathbf{x} \in \Omega \setminus \Omega^m \end{cases} \quad (15)$$

where  $\Omega^m \rightarrow$  part of the domain occupied by the material.

For sizing problem:

$$\mathbb{C}_{ijkl}(\mathbf{x}) = h(\mathbf{x}) \bar{\mathbb{C}}_{ijkl} \quad (16)$$

$$\int_{\Omega} \chi(\mathbf{x}) d\mathbf{x} = V_f \quad (17)$$

$$\& \int_{\Omega} h(\mathbf{x}) d\mathbf{x} = V_f. \quad (18)$$

where  $h(x)$  is a sizing function.

Traditionally shape design problems are initiated in the following manner:

$$\text{Ref domain } : \Omega_0 \in \mathbb{R}^3 \quad (19)$$

$$\underline{\phi} : \Omega_0 \rightarrow \phi(\Omega_0) \text{ is a diffeomorphism.} \quad (20)$$

$$L(\mathbf{v}) = \int_{\Omega_0} \mathbf{f} \cdot \mathbf{v} |det(D\underline{\phi}^{-1})| d\mathbf{x} + \int_{\partial\Omega_t} \mathbf{t} \cdot \mathbf{v} |det(D\underline{\phi}^{-1})| ds \quad (21)$$

$$\begin{aligned}
a_E &= \int_{\Omega} \mathbb{C}_{ijkl}(\mathbf{x}) \varepsilon_{kl}(\mathbf{v}) \varepsilon_{ij}(\mathbf{v}) d\mathbf{x} \\
&= \int_{\Omega_0} \mathbb{C}_{ijkl} \varepsilon_{kl}(\mathbf{v}) \varepsilon_{ij}(\mathbf{v}) |det(D\phi^{-1})| d\mathbf{x}
\end{aligned} \tag{22}$$

Now,

$$\begin{aligned}
\mathbb{C}_{ijkl} \varepsilon_{kl} &= \mathbb{C}_{ijkl} \frac{1}{2} (u_{k,l} + u_{l,k}) \\
&= \frac{1}{2} \mathbb{C}_{ijkl} u_{k,l} + \frac{1}{2} \mathbb{C}_{ijlk} u_{l,k} \\
&= \mathbb{C}_{ijkl} u_{k,l}
\end{aligned} \tag{23}$$

$$\begin{aligned}
a_E &= \int_{\Omega_0} \mathbb{C}_{ijkl} u_{k,l}(\mathbf{u}) u_{i,j}(\mathbf{v}) |det(D\phi^{-1})| d\mathbf{x} \\
&= \int_{\Omega_0} \mathbb{C}_{ijkl} \frac{\partial u_k}{\partial \mathbf{x}_m} (D\phi^{-1})_{ml} \frac{\partial u_i}{\partial \mathbf{x}_p} (D\phi^{-1})_{pj} |det(D\phi^{-1})| d\mathbf{x}
\end{aligned} \tag{24}$$

$$\Rightarrow \mathbb{C}_{ijkl} (D\phi^{-1})_{ml} (D\phi^{-1})_{pj} |det(D\phi^{-1})| = \bar{\mathbb{C}}_{ipkm} \tag{25}$$

$$\bar{\mathbb{C}}_{ijkl} = \mathbb{C}_{ipkm} (D\phi^{-1})_{lm} (D\phi^{-1})_{jp} |det(D\phi^{-1})| \tag{26}$$

Treating  $\phi$  as a design variable is tedious.

### 3.3 Homogenization method

$$E_{ijkl}^\varepsilon(\mathbf{x}) = E_{ijkl}(\mathbf{x}, \mathbf{y}), \quad \mathbf{y} = \frac{\mathbf{x}}{\varepsilon} \tag{27}$$

The tensor  $E_{ijkl}^\varepsilon$  is a material constant which satisfies the symmetry condition and is assumed to satisfy strong ellipticity condition for every  $\mathbf{x}$ .

$$\Rightarrow E_{ijkl}^\varepsilon = E_{jikl}^\varepsilon = E_{ijlk}^\varepsilon = E_{klji}^\varepsilon \tag{28}$$

$$E_{ijkl}^\varepsilon(\mathbf{x}) \mathbf{X}_{ij} \mathbf{X}_{kl} \geq m \mathbf{X}_{ij} \mathbf{X}_{ij} \quad \text{for some } m > 0 \text{ \& \forall } \mathbf{X}_{ij} = \mathbf{X}_{ji} \tag{29}$$

Let the domain  $\Omega$  has a boundary  $\Gamma$ . Let  $\mathbf{f}$  be the body force acting on  $\Omega$  and  $\mathbf{t}$  be the traction acting on  $\Gamma_t$  part of the boundary  $\Gamma$ . Also, let  $\Gamma_D$  be the part of boundary on which displacement is defined. Then the displacement  $\mathbf{u}^\varepsilon$  can be obtained as the solution to the following minimization problem

$$\min_{\mathbf{v}^\varepsilon \in U} F^\varepsilon(\mathbf{v}^\varepsilon), \tag{30}$$

$$\tag{31}$$

where  $F^\varepsilon$  is total potential energy given as

$$F^\varepsilon(\mathbf{v}^\varepsilon) = \frac{1}{2} \int_{\Omega} E_{ijkl}^\varepsilon \varepsilon_{kl}(\mathbf{v}^\varepsilon) \varepsilon_{ij}(\mathbf{v}^\varepsilon) dx - \int_{\Omega} \mathbf{f} \cdot \mathbf{v}^\varepsilon dx - \int_{\Gamma_t} \mathbf{t} \cdot \mathbf{v}^\varepsilon ds \tag{32}$$

and  $\mathcal{U}$  is the set of admissible displacements defined such that

$$\mathcal{U} = \{\mathbf{v} = v_i \mathbf{e}_i : v_i \in H^1(\Omega) \text{ and } \mathbf{v} \in \mathcal{G} \text{ on } \Gamma_D\} \quad (33)$$

where  $\mathcal{G}$  is set of displacement defined along the boundary  $\Gamma_D$ .  
Let

$$\mathbf{v}^\varepsilon(\mathbf{x}) = \mathbf{v}_0(\mathbf{x}) + \varepsilon \mathbf{v}_1(\mathbf{x}, \mathbf{y}), \quad y = \frac{\mathbf{x}}{\varepsilon} \quad (34)$$