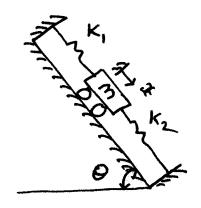
ASSIGNMENT 2 : SOLUTIONS

Q1 TEXT BOOK EXERCISE PROBLEM 2.9



NOTE THAT IN STATIC EQUILIBRIUM POSITION THERE ARE FORCES
IN THE SPRINGS KI & K2. THESE FORCES BALANCE THE
GRAVITATIONAL WEIGHT OF THE MASS.

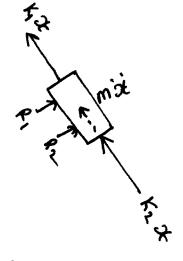
IF WE CONSIDER IF AS THE DISPLACEMENT MEASURED FROM
THE STATIC EQUILIBRIUM POSITION, THEN GRAVITY CAN
BE IGNORED. BECAUSE, SPRING FORCES CANCEL GRAVITY AT EQUILIBRIUM!

FBO ABOUT STATIC EQUILIBRIUM

SEFX :0 (DIALENBERT)

=) -ma -k2 a - k1 a = 0

NOTE THAT THE REACTIONS FROM THE ROLLERS, RIBRZ, BO NOT APPEAL AS WE ARE SUMMING FORCES ALONG Z-DIRECTION WHICH IS PERPENDICULAR TO RIBRA.

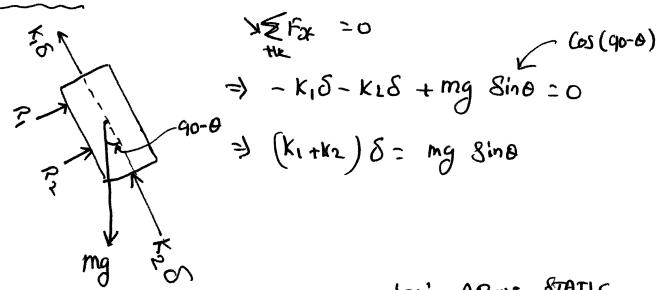


HOW DOES GRAVITY CANCEL ? (EXTRA HATERIAL) I

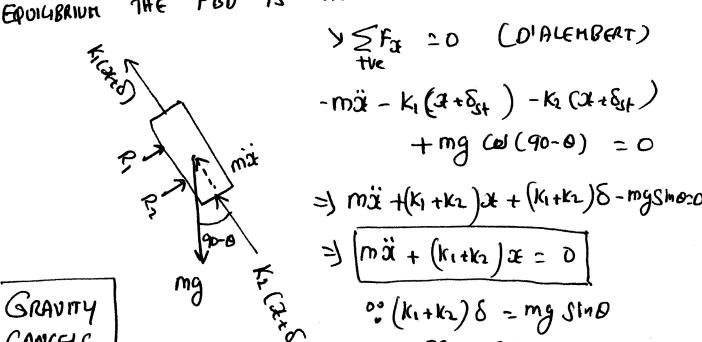
Q1) WE CAN INCLUDE GRAVITY AND SHOW THAT IT CANCELS, AS WE SAW IN THE TUTORIAL.

LET US PARST ESTABLISH STATIC EQUILIBRIUM CONDITION. IF THE SPILNES DISPLACE BY St FROM THEIR UNSTRETCHED CONFIGURATION, WE HAVE THE POLLOWING FBO IN STATICS.

STATICS FBD



FOR A DYNAMIC DISPLACEMENT OF '24' ABOUT STATIC Equilibrium THE FBO IS AS FOLLOWS.



CANCELS

FROM STATICS!

(2)

QI) TEAT BOOK EXERCISE 2.10

WE CAN MODEL THE WIRE ROPE AS A LINEAR ELASTIC SPRING. FROM PAGE 46 OF NOTES:

SPRING CONSTANT FOR WIRE OF LENGTH 30:

$$K_1 = \frac{A_1 E_1}{L_1}$$

AL= AREA OF CIS = TI (0.05)2 GIVEN DIANTER= 0.05"

E1 = YOUNG'S MOONLYS OF STEEL = 30 X106 10/in2

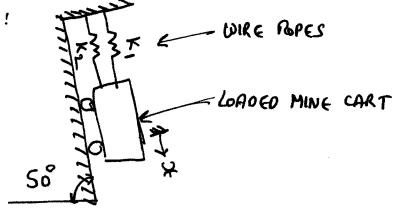
4 = 30' = 30 x12'

 $\therefore K_{1} = \frac{\pi}{4} (0.05)^{2} \times 30 \times 10^{6}$ $= \frac{163.625}{30 \times 12}$

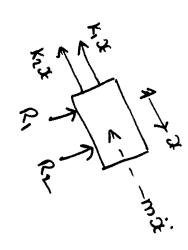
SIMILARLY SPRING CONSTANT FOR WIRE OF LEWGTH 25':

 $K_1 = \frac{A_2 E_2}{L_2} = \frac{\pi}{4} (0.05)^L \times 30 \times 10^6 = 196.35 \text{ lb/in}$

SO OUR MODEL IS!



(91) WE CAN FORHULATE THE EQUATION OF MOTION FROM THE
FOLLOWING FOO (VERY SIMILAR TO PROBLEM 2.4 SOLVED BEFORE)



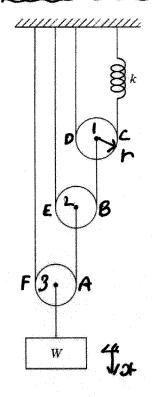
GRAVITY CANCELS AND HENCE IGNORED.

 $4 \le F_{x} = 0$ (DIALENBERT) = $-m\ddot{x} - K_1 x - K_2 x = 0$

GIVEN m = 5000 lbKNOWN $K_{1} = 163.625 \text{ lb/lin}$ $K_{2} = 196.35 \text{ lb/lin}$

NOTE THAT THE ABOVE PROBLET IS VERY SIMILAR TO THE PROBLEM 2.9 SOLVED EARLIER. CONVINCE YOURSELF THAT GRAVITY DOES CANCEL BY POLLOWING THE SAME PROCEDURE AS WE DID IN PROBLEM 2.9.

Q1) TEXT BOOK EXERCUSE PROBLEM 2.14



IF THE WEIGHT MOVES POWN BY A DISTANCE I, THEN A MOVES BY 21, B MOVES BY 42, AND C MOVES BY 82. THIS IS FROM KINEMATICS OF NO-SUP (TOPICI, PAGE 8 IN YOUR NOTES)

NOTE!

12 RAQUS OF PULLEYS

PULLEY INERTIA IGNORED

7 \(\frac{1}{2} \) = \(\) \(

PULLEY 2: F3 1 F2 = 16KX

4 2 MA = 0

TH

FL r- F3+=0 => F3 = 16KA

PULLEY 8:
$$f_4 = 32 \text{ K} \times 4 \times 10^{-20}$$

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$$7 \leq F_{y} = 0 \Rightarrow F_{5} + F_{4} - F_{6} = 0$$

the $\Rightarrow F_{6} = F_{5} + F_{4}$
 $\Rightarrow F_{6} = 64 \text{ KeV}$

FBO OF WEIGHT:

$$\int_{0}^{\infty} f_{6} = 64 \text{ K} \text{ A}$$

$$\int_{0}^{\infty} f_{6} = 64 \text{ K} \text{$$

GRAVITATIONAL WEIGHT BALANCES INITIAL TENSIONS SPRING TORCE.

SO GRAVITY IGNORED IN THE ABOVE FBD. NOTE THE

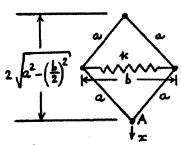
INERTIA OF WEIGHT CAN'T BE IGNORED!L

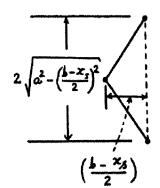
8 = ACCELERATION DUE TO GRAVITY.

NOTE: A CAREFUL CONSTRUCTION OF FBDS IS A MUST TO ______ GET THE ABOVE RIGHT. PRACTISE WILL HELP YOU.

TEXTBOOK EXERCISE PROBLEM 1.11

92)





Potential energy equivalence gives $\frac{1}{2} k_{eq} x^2 = \frac{1}{2} k x_s^2$

$$\begin{aligned}
\kappa_{e_g} &= \kappa \left(\frac{x_g}{x} \right)^2 \\
\text{But} \quad x &= 2 \left[\sqrt{a^2 - \left(\frac{b - x_g}{2} \right)^2} - \sqrt{a^2 - \left(\frac{b}{2} \right)^2} \right] \\
&= 2 \sqrt{a^2 - \left(\frac{b}{2} \right)^2} \left[\left\{ \frac{a^2 - \left\{ \frac{b}{2} \left(1 - \frac{x_g}{b} \right) \right\}^2}{a^2 - \left(\frac{b}{2} \right)^2} \right\}^{\frac{1}{2}} - 1 \right] \\
&= 2 \sqrt{a^2 - \frac{b^2}{4}} \left[\left\{ \left(\frac{a^2 - \frac{b^2}{4} - \frac{x_g^2}{4} + \frac{b - x_g}{2}}{a^2 - \frac{b^2}{4}} \right) \right\}^{\frac{1}{2}} - 1 \right] \\
&= 2 \sqrt{a^2 - \frac{b^2}{4}} \left[\left\{ 1 - \frac{x_g^2 + x_g}{a^2 - \frac{b^2}{4}} + \frac{b - x_g}{2}}{a^2 - \frac{b^2}{4}} \right\}^{\frac{1}{2}} - 1 \right]
\end{aligned}$$

Using the relation $(1+\theta)^{1/2} \simeq 1+\frac{\theta}{2}$, we obtain

$$x = 2 \left(a^2 - \frac{b^2}{4}\right)^{1/2} \left[1 + \frac{b^2 x_3}{4(a^2 - \frac{b^2}{4})} - 1\right] = \frac{b^2 x_4}{2(a^2 - \frac{b^2}{4})^{1/2}}$$

:
$$k_{e_0} = k \left(\frac{x_s}{x}\right)^2 = 4k \left(\frac{a^2 - \frac{b^2}{4}}{b^2}\right) = k \left(\frac{4a^2 - b^2}{b^2}\right)$$

(b) Here
$$x = x_s$$
 (spring deflection)
 $\therefore k_{eg} = k$

Q2) TEXT BOOK EXERCISE PROBLEM 1.10

For simply supported beam, for load at middle, $k_1 = \frac{48 \, \text{EI}}{l^3} = \frac{48 (2.06 \, \text{x 10}^{11}) (10^4)}{8}$ $= 12.36 \, \text{x 10}^7 \, \text{N/m} \quad \text{where } I = \frac{1}{12} \left(1.2 \right) \left(0.1 \right)^3 = 10^4 \, \text{m}^4.$ $\delta_1 = \text{original deflection} = \frac{mg}{k_1} = \frac{500 \, \text{x} \, 9.81}{12.36 \, \text{x 10}^7} = 396.8447 \, \text{x 10}^7 \, \text{m}$ When spring k is added, kep = $\frac{1}{12} \, \frac{1}{12} \, \frac{1}{1$

(a) New deflection =
$$\frac{mg}{k_{eg}} = \frac{\delta_1}{4}$$
; $k_{eg} = \frac{4 mg}{\epsilon_1} = 4 k_1$
 $\therefore k = 3 k_1 = 37.08 \times 10^7 \text{ N/m}$

(b) New deflection =
$$\frac{mg}{keg} = \frac{\delta_1}{2}$$
; $keg = \frac{2mg}{\delta_1} = 2k_1$
 $k = k_1 = 12.36 \times 10^7 \text{ N/m}$ = $k + k_1$

(c) New deflection =
$$\frac{mg}{keg} = \frac{3}{4} \delta_1$$
; $keg = \frac{4 mg}{3 \delta_1} = \frac{4}{3} k_1$
 $\therefore k = \frac{1}{3} k_1 = 4.12 \times 10^7 N/m = k + k_1$

Q2) TEATBOOK EXERCISE PROBLEM 1. 29

Assume small angles
$$\theta_{1}$$
 and θ_{2} ; $\theta_{2} = \left(\frac{p_{1}}{t_{2}}\right)\theta_{1}$
 $x_{1} = \text{horizontal displacement of c.G. of mass } m_{1} = \theta_{1} r_{1}$
 $x_{2} = \text{vertical displacement of c.G. of mass } m_{2} = \theta_{2} r_{2} = \frac{p_{1}\theta_{1} r_{2}}{p_{2}}$
 $y_{1} = \text{horizontal displacement of springs } k_{1} \text{ and } k_{2} = \theta_{1} \left(r_{1} + l_{1}\right)$
 $y_{2} = \text{vertical displacement of springs } k_{3} \text{ and } k_{4} = \theta_{2} l_{2} = \frac{p_{1}l_{1}\theta_{1}}{p_{2}}$

Equivalence of kinetic energies gives

$$\frac{1}{2} \text{ Jeq}(\hat{\theta}_{1})^{2} = \frac{1}{2} \text{ Ji}(\hat{\theta}_{1})^{2} + \frac{1}{2} \text{ Ji}(\hat{\theta}_{2})^{2} + \frac{1}{2} \text{ m_{1}}(\hat{x}_{1})^{2} + \frac{1}{2} \text{ m_{2}}(\hat{x}_{2})^{2}$$

$$\therefore \text{ Jeq} = J_{1} + J_{2} \left(\frac{p_{1}}{p_{2}}\right)^{2} + m_{1} r_{1}^{2} + m_{2} r_{2}^{2} \left(\frac{p_{1}}{p_{2}}\right)^{2}$$

Equivalence of potential energies gives

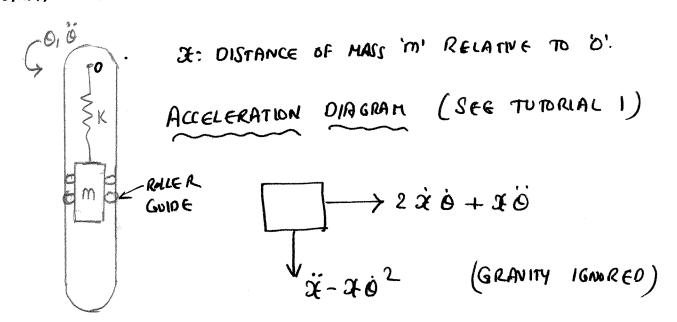
$$\frac{1}{2} \text{ keq } \theta_{1}^{2} = \frac{1}{2} \text{ k_{12}} \text{ y}_{1}^{2} + \frac{1}{2} \text{ k_{34}} \text{ y}_{2}^{2} + \frac{1}{2} \text{ k_{41}} \theta_{1}^{2} + \frac{1}{2} \text{ k_{42}} \theta_{2}^{2}$$

with $k_{12} = k_{1} + k_{2}$, $k_{34} = k_{3} k_{4} / (k_{3} + k_{4})$

$$J_{1} = \theta_{1}(r_{1} + l_{1}), J_{2} = \frac{p_{1}l_{2}\theta_{1}}{p_{2}} \text{ and } \theta_{2} = \frac{p_{1}\theta_{1}}{p_{2}} \cdot \frac{p_{2}^{2}}{p_{2}^{2}}$$

$$\therefore \text{ keg} = (k_{1} + k_{2}) \left(\beta_{1} + l_{1}\right)^{2} + \left(\frac{k_{3}}{k_{3}} \frac{k_{4}}{k_{3}}\right) \frac{p_{1}^{2} l_{2}^{2}}{p_{2}^{2}} + k_{11} + k_{12} \cdot \frac{p_{1}^{2}}{p_{2}^{2}}.$$

(43) TUTORIAL I PROBLEM (ii) IN QUESTIONI: REPATING SPRING MASS SYSTEM



FORCES IN FBO (D'ALENBERT)

$$R_{1} \xrightarrow{KX} R_{3}$$

$$R_{2} \xrightarrow{R_{3}} R_{4}$$

$$\lim_{x \to \infty} (\ddot{x} - x \dot{o}^{2})$$

$$\begin{array}{l}
+ \sqrt{2} f_{2} = 0 \\
- Kx - m(\ddot{x} - x \dot{o}^{2}) = 0
\end{array}$$

$$= \sqrt{m\ddot{x} + (K - m\dot{o}^{2})} x = 0$$

NOTE EFFECTIVE STIFFNESS K-MÖZ DEPENDS ON RETATIONAL SPEED!
HORIZONTAL EQUILIBRIUM DETERMINES NET REACTION FROM
ROLLAR GUIDES!