

MECH 467/589 -Final Exam 12:00-2:30pm (December 7, 2019) Prof. Y. Altintas
CLOSED BOOK EXAM: No phone/calculators. PUT A BOX AROUND YOUR ANSWERS.
WRITE YOUR ANSWERS IN A LEGIBLE WAY TO AVOID LOOSING MARKS

1. Open loop dynamics of a process is defined by the following differential equation where $u(t)$ is the input and $y(t)$ is the output to the physical, uncontrolled process:

$$\frac{d^2y}{dt^2} + \frac{dy}{dt} - 12y = \frac{du}{dt} + 2u$$

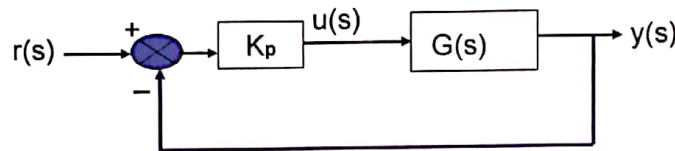
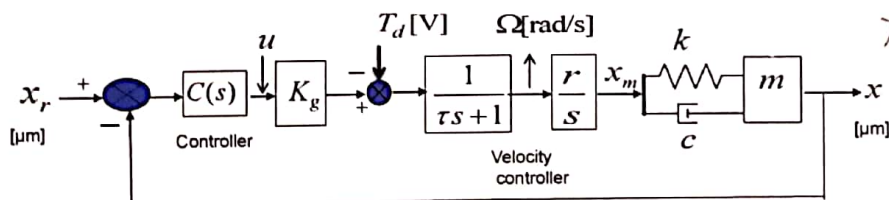


Figure 1: Block diagram of a closed loop system with a proportional controller.

- (5) Obtain the transfer function of the open loop dynamics of the process
 - ★ (10) Plot the Bode diagram of the open loop system approximately (i.e. $\omega = 1, 10, 100$)? (You can use logarithmic scale in dB or just a linear scale)
 - ★ (10) If a proportional controller is used to control the closed loop system, find the value of proportional gain K_p for the critically stable closed loop system? What is the range of K_p to ensure the stability?
2. A ball screw system is driving a table with a single degree of flexibility between the motor and ball screw assembly system. The position command is x_r , the position of the motor shaft is x_m and the table position is x . The block diagram of the system is given in the figure. The equation of the motion for the flexible system is: $m\ddot{x} = -c(\dot{x} - \dot{x}_m) - k(x - x_m)$ which has a natural frequency of $\omega_n = \sqrt{k/m}$. The controller is a low pass filter with a gain $C(s) = K_p\omega_n/(s + \omega_n)$.



★ $C(s) = K_p \frac{\omega_n}{s + \omega_n}$
 Low pass filter

Figure 2: Block diagram of a closed loop system with a flexible table attachment. Note: $C(s) = K_p s/(s + \omega_n)$

- ✓ (10) Express the frequency response of the loop transfer function of the system (Magnitude and Phase)
 - ✓ (10) Express the closed loop transfer functions ($G_x(s), G_d(s)$) of the system including disturbance torque ($x(s) = G_x(s)x_r(s) + G_d(s)T_d(s)$)
 - (10) What is the steady error for a ramp position input ($x_r(t) = ft$) and step disturbance input ($T_d(s) = T_0$)
3. A plant's open loop transfer function is given as $G(z) = \frac{z^{-2}B(z^{-1})}{A(z^{-1})} = \frac{z^{-2}b}{1+a_1z^{-1}}$.
- (15)) Design a pole placement controller to achieve a damping ratio of ζ and natural frequency ω_n ? ($R(z^{-1}) = 1 + r_1z^{-1} + r_2z^{-2} + \dots + r_pz^{-p} \rightarrow \deg(R) = p$; $S(z) = s_0 + s_1z^{-1} + s_2z^{-2} + \dots + s_fz^{-f} \rightarrow \deg(S) = f$; $t_0 = ?$)
 - (10) Express the control law at each time interval, i.e. $u(k) = ??x(k) + ??x(k-1) + \dots + ??u(k-1) + ??u(k-2) + \dots$

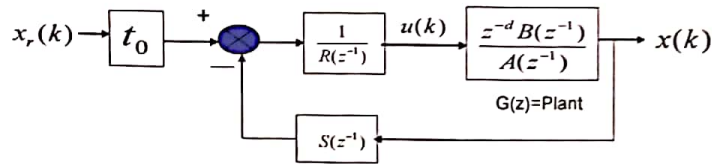


Figure 3: A pole placement controller system.

Notes:

Diophantine casuality equilibrium - $p + f + 1 = d + n + f$; $p + f + 1 = m + p$; mapping of a pole from s to z plane: $z^{-1} = e^{-sT}$ where T is the control time interval.

Pole placement principle: $G_{cl}(z) = \frac{x(k)}{x_r(k)} = \frac{z^{-d} B t_0}{A R + z^{-d} B S} \equiv \frac{z^{-d} B_m}{A_m}$ where A_m is the characteristic equation with the desired damping ratio ζ and natural frequency ω_n .

4. (20) Design a linear interpolator between points $P(0,0)$ and $P(x_2, y_2)$. The cruising tangential velocity is f_c , the acceleration and deceleration are the same $A = A$ and $D = -A$. The initial ($f(0) = 0$) and final ($f_e(t_3) = 0$) velocities are zero, and the interpolation time interval is T_i . Total travel time is $T = T_1 + T_2 + T_3$. Express $T_1, T_2, T_3; l(k) = ??, \Delta l(k) = ?, \Delta x(k) = ?, \Delta y(k) = ? \rightarrow k = 1, 2, \dots, N$. for each zone

$$\left\{ \begin{array}{ll} \text{Zone I} & a(t) = A ; \quad f(t) = At; \quad l(t) = \int f(t) dt \quad 0 \leq t \leq t_1 \rightarrow T_1 = t_1 \\ \text{Zone II} & a(t) = 0 ; \quad f_c; \quad l(t) = l(t_1) + \int f_c dt \quad t_1 \leq t \leq t_2 \rightarrow T_2 = t_2 - t_1 \\ \text{Zone III} & a(t) = -A ; \quad f_c - At; \quad l(t) = l(t_2) + \int f(t) dt \quad t_2 \leq t \leq t_3 \rightarrow T_3 = t_3 - t_2 \end{array} \right\}$$

Additional Formulas:

$x(t)$	1	e^{-at}	t	Final value theory
$X(s)$	$\frac{1}{s}$	$\frac{1}{s+a}$	$\frac{1}{s^2}$	Continuous time domain $\lim_{t \rightarrow +\infty} f(t) = \lim_{s \rightarrow 0} sF(s)$
$X(z)$	$\frac{1}{1-z^{-1}}$	$\frac{1}{1-e^{-aT}z^{-1}}$	$\frac{Tz^{-1}}{(1-z^{-1})^2}$	Discrete time domain $\lim_{t \rightarrow +\infty} f(t) = \lim_{z \rightarrow 1} (1-z^{-1})F(z)$