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**THERMAL EVOLUTION OF URANUS AND NEPTUNE WITH
CONDENSATION-INHIBITED CONVECTION**

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Contents

List of Figures	v
1 Introduction	1
1.1 Relevant Thermodynamics	2
1.1.1 Temperatures	2
1.1.2 Means of Energy Transport and Gradients	3
1.1.3 Condensation in Hydrogen Dominated Atmospheres	4
1.2 Prior Work	5
2 Model	8
2.1 Basic Equations	8
2.2 Inclusion of Moist Adiabat Within Outer Envelope	11
2.3 Temperature Jump Across the Water Condensation Zone	14
2.4 Energy Conservation and Thermal Evolution of Model	16
3 Results	17
3.1 Condensation-inhibited Convection	17
3.2 Formation of Radiative Zone	18
3.3 Thermal Evolution of Uranus and Neptune	20
4 Discussion and Conclusions	28

List of Figures

2.1	A Standard Interior Structure Model	10
2.2	Interior Structure for Moist Adiabat	12
2.3	A Standard Interior Structure Model	13
3.1	Inhibition of convection on Uranus	19
3.2	Impact of Radiative Layer on T10	21
3.3	Formation of Radiative Zone	23
3.4	Thermal Evolution Curves for Uranus - Adiabat Comparisons	24
3.5	Thermal Evolution Curves for Uranus - Water Vapor Concentration Comparisons	25
3.6	Thermal Evolution Curves for Neptune - Water Vapor Concentration Comparisons	26
3.7	Thermal Evolution Curves for Uranus - Radius	27
3.8	Thermal Evolution Curves for Neptune - Radius	27

1

Introduction

According to core accretion theory, planets coalesce from matter contained within their parent star's protoplanetary disk. A planet will accrete matter from the disk until the supply of matter has been exhausted, at which point, the planet will begin to cool and contract over time (Lissauer & Stevenson, 2007; Armitage, 2013). It is natural to ask, what is a planet's temperature as it cools? Are we talking about the temperature at the surface? What do we mean by surface? How does energy get transported through the planet's interior? Does it conduct, convect, radiate, or all of these? If so, where, and under what conditions? Do clouds form, and do they impact a planet's cooling trajectory? These are some of the questions that physicists attempt to answer when modeling giant planet interiors and atmospheres. The sections in this first chapter will begin by defining some of thermodynamic concepts relevant to the physics of giant planet interiors, and will close with a brief overview of the observations and prior work on interior structure models that specifically motivated this work. In Chapter 2, we'll describe a conventional model for ice giant interior structure, and how our moist-convective model differs. We present our results

in Chapter 3, describing where and when stable water condensation zones form, how they impact cooling within the interior, and their impact on thermal evolution. In Chapter 4, we discuss our conclusions and offer suggestions for future work.

1.1 Relevant Thermodynamics

1.1.1 Temperatures

There are several temperatures we are concerned with. Beginning with the effective temperature, T_{eff} , which is defined in terms of the total flux, F_P , integrated over all frequencies, ν , of a black body of the same shape and same distance as the planet (Seager, 2010):

$$F_P = \int_0^\infty F_P(\nu) d\nu = \pi \int_0^\infty B(T, \nu) d\nu = \sigma_B T_{\text{eff}}^4, \quad (1.1)$$

where σ_B is the Stefan-Boltzmann constant. Solving for T_{eff} yields

$$T_{\text{eff}} = \left(\frac{F_P}{\sigma_B} \right)^{\frac{1}{4}}. \quad (1.2)$$

The equilibrium temperature, T_{eq} , is the temperature the planet would have if it were in thermal equilibrium with its parent star. This occurs when the planet has radiated away its latent heat of formation, and the only remaining source of energy is from its star. This temperature is estimated as

$$T_{\text{eq}} = (T_{\text{eff}*}) \left(\frac{R_*}{a} \right)^{\frac{1}{2}} [f(1 - A_B)]^{\frac{1}{4}}, \quad (1.3)$$

where $T_{\text{eff}*}$ is the effective temperature of the parent star, R_* is the star's radius, and a is the planet's semi-major axis. The factor $(1 - A_B)$ is the fraction of energy from the parent star absorbed by the planet's atmosphere, A_B , being the bond albedo, which represent the

fraction of the parent star's incident energy that is reflected back into space. The factor f accounts for the planet's distribution of the radiation it receives from its parent star. We make the assumption assumption that for Neptune and Uranus that the Sun's radiation is evenly distributed throughout, and thus $f = 1$.

Finally, the intrinsic temperature, T_{int} , is the temperature that defines the flux from the planet's interior and is defined by the relation

$$T_{\text{eff}}^4 = T_{\text{eq}}^4 + T_{\text{int}}^4. \quad (1.4)$$

1.1.2 Means of Energy Transport and Gradients

How energy flows throughout a planet's interior impacts its vertical temperature structure. In this section, we'll review the common means of energy transport within a giant planet's interior.

Dry Adiabatic Gradient

Convection is a common form of energy transport within the interior of giant planets. In convecting regions, the interior is treated as a compressible gases or fluids. A parcel will compress or expand without exchanging heat with its surroundings. When there is no heat exchange, the process is said to be 'adiabatic'. As a parcel of gas rises, its temperature decreases while its volume increases. This process is known as adiabatic expansion. Conversely, if the parcel sinks, it gets warmer as its volume decreases. This process is known as adiabatic compression. These processes assume constant entropy. It is said that the temperature-pressure profile follows a dry adiabatic gradient, or dry lapse

rate (R. Kippenhahn, 2012), given by:

$$\nabla_{\text{ad}} = \left(\frac{\partial \ln T}{\partial \ln P} \right)_s \quad (1.5)$$

where s is entropy, T is the temperature, and P is the pressure. This can also be expressed in terms of temperature and altitude (Sanchez-Lavega, 2010) as:

$$\frac{dT}{dz} = \frac{-g}{C_P} = \Gamma_{\text{dry}} \quad (1.6)$$

where z is the altitude, and C_P is specific heat.

Moist Adiabatic Gradient

Of course, one must also consider the presence of condensable species. When vapor condenses into solid or liquid form, energy is released. This energy is referred to as latent heat

Radiative Gradient

1.1.3 Condensation in Hydrogen Dominated Atmospheres

On Earth, the atmosphere undergoes moist convection. As a parcel of air is lifted, it cools until it gets cold enough that water vapor condenses out, releasing latent heat of condensation which further boosts convection. This release of latent heat alters the temperature-pressure profile of the atmosphere, which now follows a moist adiabat. In addition to altering the temperature gradient, condensation may also create a gradient in mean molecular weight. For example, on Earth, moist air is lighter than dry air. H_2O vapor (molecular mass = 18 g/mol), the primary condensate in Earth's atmosphere, is lighter (not

by much) than the background air which is composed primarily of N₂ (molecular mass = 28 g/mol). When H₂O vapor abundance exceeds the saturation vapor pressure, the vapor condenses out of the atmosphere, resulting in a small vertical gradient in mean molecular weight. In Earth's atmosphere, this small gradient does not impose a significant barrier to convection. By contrast, in hydrogen dominated atmospheres such as Neptune and Uranus, the background gas is much lighter than the condensates. In this hydrogen-rich environment, when H₂O condenses out of the atmosphere, a strong vertical gradient in mean molecular weight can be established, resulting in a negative buoyancy for the convecting parcel of gas. This can create a situation where the zone in which water condenses is stable against convection (Guillot, 1995), (Friedson & Gonzales, 2017), (Leconte et al., 2017). Thus far, we have been discussing H₂O as the only condensate. However, other condensates such as NH₃ and CH₄ may impact convection as well. In this study, we only consider H₂O as the primary condensate as it likely has the largest impact. The reason for this is that if its abundance is supercritical, then it results in a larger superadiabicity (larger temperature gradient) than would be provided by either NH₃ or CH₄ (Guillot, 1995). Consideration of other condensates is planned for future work.

1.2 Prior Work

In 1965, Frank Low measured Jupiter's intrinsic temperature (Low, 1966). To explain this observation, theorists set out to expand on prior work by (Demarcus, 1958) on the theory of interior structure of solar system gas and ice giants (Hubbard, 1968; Smoluchowski, 1967; Hubbard, 1977, 1978; M. Podolak, 1991). These models assumed that the interior of giant planets are convective, meaning that heat within the planet's interior is

2

Model

2.1 Basic Equations

We begin our description of the physics of our interior structure model by assuming spherical symmetry and conservation of mass:

$$\frac{dm}{dr} = 4\pi r^2 \rho \quad (2.1)$$

where dm is the mass contained within a sphere of radius $r + dr$, and ρ is the density.

Hydrostatic equilibrium is also assumed and described by:

$$\frac{dP}{dr} = -\frac{Gm\rho}{r^2} \quad (2.2)$$

where P is the pressure and G is the gravitational constant.

We employ a three-layer interior structure, seen schematically in Figure 2.1. At the center of the planet is a core of mass, m_{core} . The core is made of pure water ice, indicated by $Z = 1$, the H₂O mass fraction. Moving outward, the inner envelope is H₂O

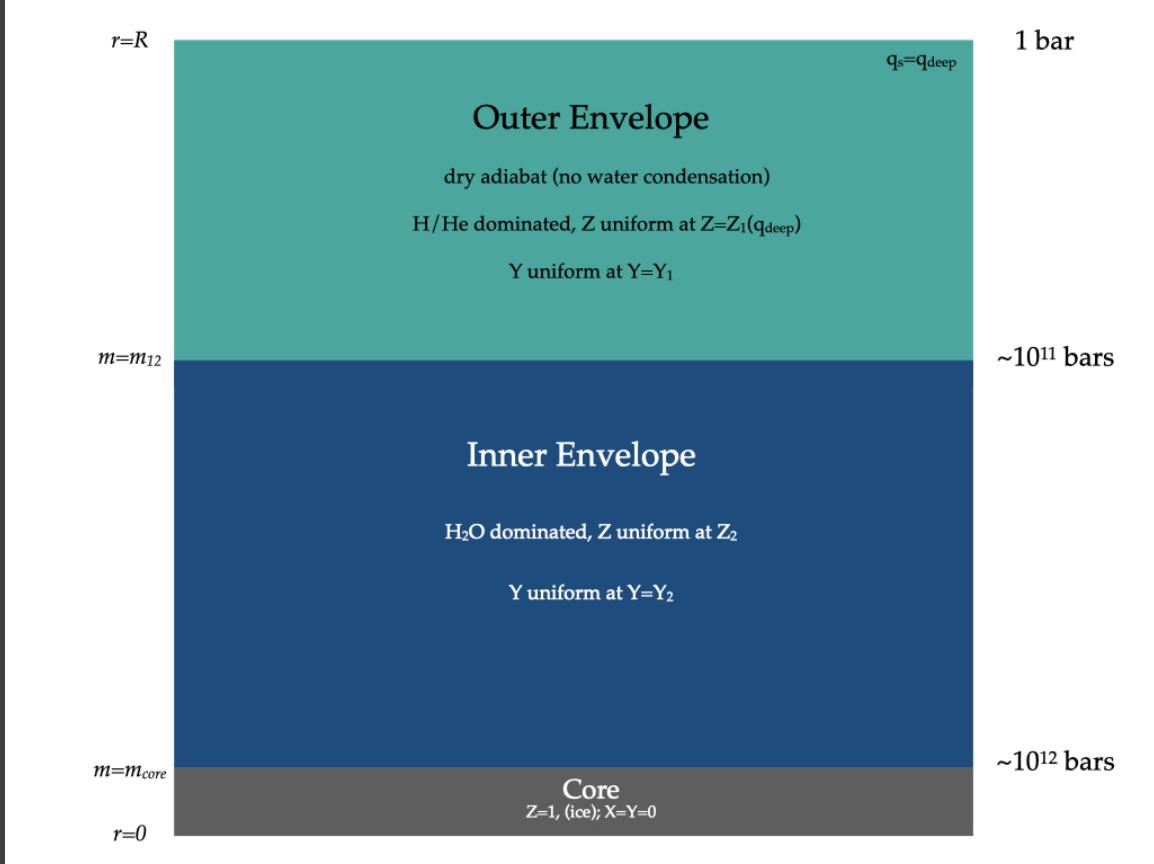


Figure 2.1: A conventional interior structure: Fully convective and dry adiabatic. In this model, the inner and outer envelopes are assumed to be well mixed, fully convective, and following a dry adiabat. The core is composed of water ice. The inner envelope is water dominated, with uniform concentrations of hydrogen, helium, and water; whereas, the outer envelope is hydrogen and helium dominated with trace amounts of water. The atmosphere extends beyond 1 bar, but pressures down to 1 bar are sufficient to capture the formation and impact of the water condensation zones investigated here.

Finally, beyond the outer envelope is the atmosphere. Atmospheres regulate how quickly the energy within a planet's interior can radiate into space. When modeling the thermal evolution of gas and ice giants, it has long been recognized that model atmospheres constitute an outer boundary condition for interior structure models, providing key inputs

that impact cooling times for interior structure models (Graboske et al., 1975; Fortney et al., 2011). Specifically, model atmospheres allow us to link the planet's T_{int} and T_{eff} to the model's surface gravity, g , and its T_1 or T_{10} . Our work utilizes the (Fortney et al., 2011) model atmosphere.

2.2 Inclusion of Moist Adiabat Within Outer Envelope

Our interior structure model modifies the conventional structure described in Section 2.1 by adding a moist adiabatic layer to the outer envelope, which under favorable conditions, allows for the condensation of H₂O. Gases condense at sufficiently low temperatures or high pressures. Condensation of a gas is characterized by its saturation vapor pressure, which derives from the Clausius-Clapeyron equation (Lavega, 2011). The saturation vapor pressure, P_{sat} , is given by

$$P_{\text{sat}}(T) = P_{\text{sat}}(T_0)e^{-\frac{L+C_p T_0}{R_{\text{vap}}}(\frac{1}{T}-\frac{1}{T_0})-\frac{C_p}{R_{\text{vap}}}\ln\frac{T}{T_0}} \quad (2.8)$$

where $T_0 = 273.16K$, and R_{vap} is the gas constant for the condensable species. When the partial pressure of a gas, P_{gas} , is less than P_{sat} , the parcel of gas is 'subsaturated'. When $P_{\text{gas}} = P_{\text{sat}}$, the gas is 'saturated'. And, when $P_{\text{gas}} > P_{\text{sat}}$, the parcel is 'supersaturated'. Every condensable species has its own saturation vapor pressure. We define the moist adiabat as (Lavega, 2011)

$$\nabla_{\text{moist}} = \left(1 + \frac{\frac{x_{\text{vap}} L}{R_{\text{vap}} T}}{\nabla_{ad} + \frac{L^2}{R_{\text{vap}}^2 T^2}}\right) \quad (2.9)$$

where

$$x_{\text{vap}} = \frac{P_{\text{sat}}}{P} \quad (2.10)$$

and

$$\frac{dT}{dP} = \frac{T}{P} \nabla_{\text{moist}} \quad (2.11)$$

and the gradient of the water vapor mole fraction is given by

$$\frac{dx_{\text{vap}}}{dP} = \frac{x_{\text{vap}} L}{R_{\text{vap}} T^2} \frac{dT}{dP} - \frac{x_{\text{vap}}}{P} \quad (2.12)$$

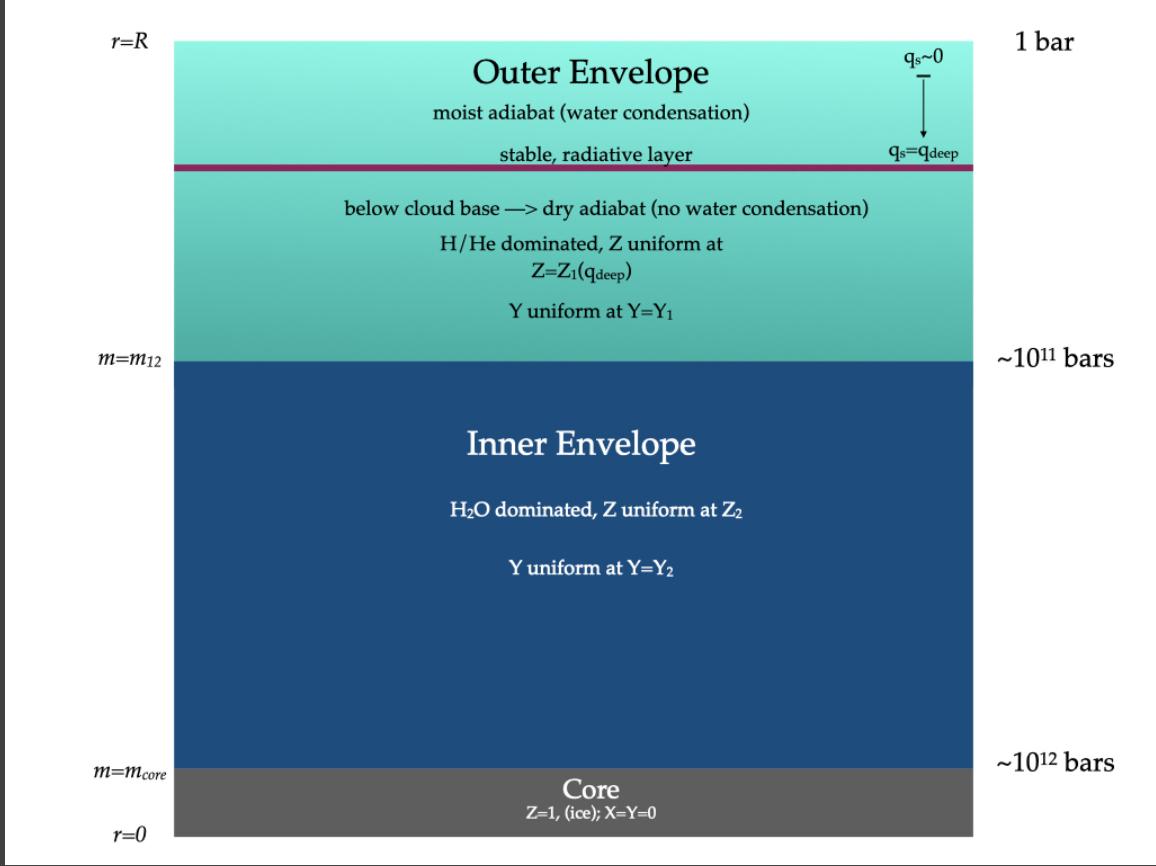


Figure 2.2: The structure for moist adiabatic interior, allowing for condensation-inhibited convection. In this model, a stable water condensation zone may form. The red horizontal line indicates the radiative zone (water condensation zone). The pressure and temperature at the base of the condensation zone is set by the condition that x_{vap} has reached the deep value $x_{\text{vap}}^{\text{deep}}$. Below the condensation zone, the temperature and pressure follow a dry adiabat.

In Figure 2.3, we compare pressure-temperature and pressure-xvap profiles that

follow a dry adiabatic lapse rate, a moist adiabatic lapse rate, and a moist adiabatic lapse rate containing a radiative layer at some depth. The profile of the moist adiabatic lapse rate is cooler at depth than either of the other two profiles. However, the presence of a stable radiative layer results in a warmer interior. These profiles assume $q_{\text{deep}} = 0.25$ and $T_1 = 150K$, which is approximately when the onset of condensation-inhibited convection occurs, as will be shown in Chapter 3. When the pressure-temperature profile follows a dry adiabat, the vapor mole fraction, x_{vap} , is constant.

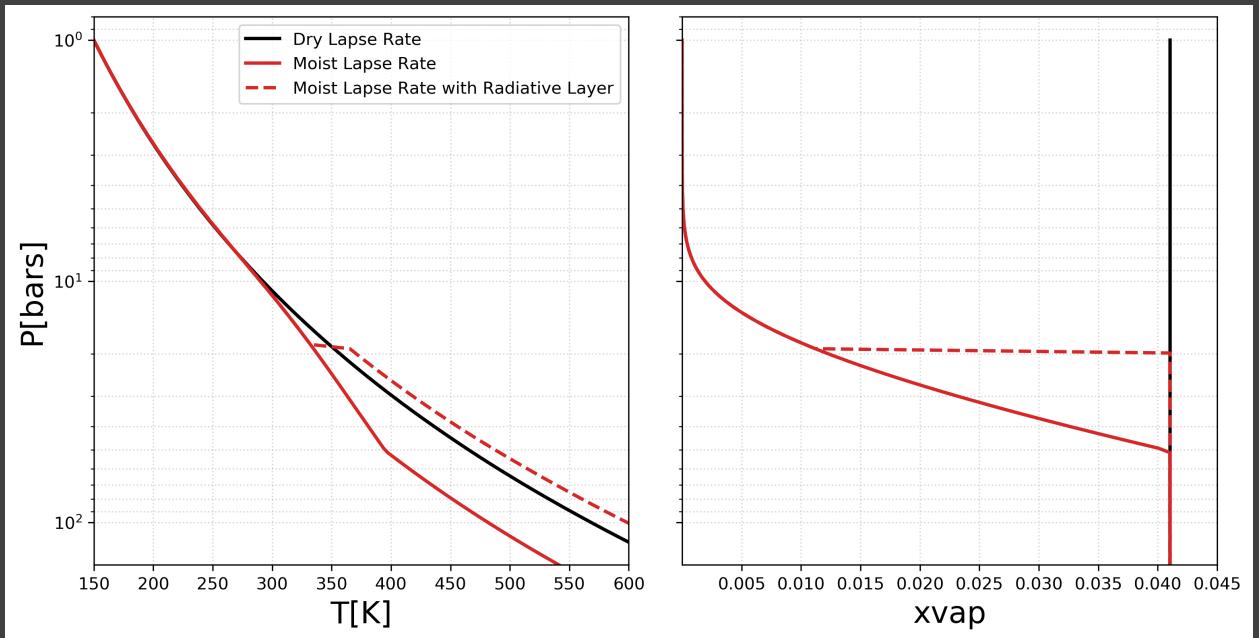


Figure 2.3: The solid red line is the profile following moist adiabatic lapse rate. The solid black line is following the the dry adiabatic lapse rate. The dashed red line is following a moist adiabatic lapse rate with the addition of a stable radiative layer (water condensation zone).

If condensation occurs, our model assumes that it may be stable against convection if a fast rain-out occurs such that the vertical gradient in mean molecular weight is large enough to counteract the positive buoyancy of the parcel of gas (Leconte et al., 2017) (Friedson & Gonzales, 2017). In this scenario, convection is interrupted when α is negative,

where α (Friedson & Gonzales, 2017) is given by:

$$\alpha = 1 + \xi(q_s L / R_{\text{vap}} T_0) \quad (2.13)$$

where R_{vap} is the gas constant for the vapor (water), T_0 is the local temperature, L is the latent heat of vaporization for water, q_s is the saturation specific humidity, and ξ is given by $\xi = \frac{1}{\epsilon} - 1$, where ϵ is the ratio of the molecular weight of vapor to the mean molecular weight of dry atmosphere. In our case, $\xi \approx -0.872$. When α is negative, the vertical gradient in molecular weight results in a stabilizing effect, overwhelming the effects due to latent heat release.

2.3 Temperature Jump Across the Water Condensation Zone

In a layer in which alpha, Eqn. 2.13, becomes negative, convection is interrupted. In the limit that H₂O rains out quickly, (Friedson & Gonzales, 2017; Leconte et al., 2017) it has been shown that radiative diffusion is responsible for heat transport, with the temperature gradient across the zone following a radiative temperature gradient (R. Kippenhahn, 2012)

$$\left(\frac{dT}{dP} \right)_{\text{rad}} = \frac{T}{P} \nabla_{\text{rad}} = \frac{T}{P} \times \frac{3}{16} \frac{\kappa_R P}{g} \frac{T_{\text{int}}^4}{T^4} \quad (2.14)$$

Due to the large opacities, κ_R , that are typical of giant planet interiors, the radiative gradient is significantly larger than the either the dry adiabatic gradient or moist adiabatic gradient. This radiative gradient results in a sharp temperature increase as seen with the dashed-red curve in Figure 2.3. Since our model has a finite radial resolution, it is unable to spatially resolve the temperature change. Instead, the model treats the water condensation

zone (a thin, stable, radiative layer) as a discontinuous increase in temperature. Nevertheless, this radiative zone corresponds to a continuum of temperatures that is governed by

$$T(P) = T_{\text{top}} + \int_{P_{\text{top}}}^{P_{\text{base}}} \left(\frac{dT}{dP} \right)_{\text{rad}} dP \quad (2.15)$$

with P_{top} and T_{top} denote the pressure and temperature at the top of the stable water condensation zone, and P_{base} represents the bottom of the zone. The radiative temperature gradient across the water condensation zone is nearly constant, so that Eqn. 2.15 simplifies to

$$T_{\text{base}} \equiv T(P + \Delta P) = T_{\text{top}} + \left(\frac{dT}{dP} \right)_{\text{rad}} \Delta P \quad (2.16)$$

where ΔP is the extent of the pressure-space of the water condensation zone (radiative layer), given by

$$\Delta P \equiv P_{\text{base}} - P_{\text{top}} = \frac{P_{\text{sat}}(T_{\text{base}})}{x_{\text{vap}}^{\text{deep}}} - P_{\text{top}}. \quad (2.17)$$

Within the condensation zone, the vapor mole fraction, x_{vap} is equal to the saturated vapor mole fraction:

$$x_{\text{vap}}(P, T) = x_{\text{vap}}^{\text{sat}}(P, T) = \frac{P_{\text{sat}}(T)}{P}, \quad P < P_{\text{base}}. \quad (2.18)$$

The base of the condensation zone is set by the condition that x_{vap} has reached the deep value $x_{\text{vap}}^{\text{deep}}$:

$$x_{\text{vap}}^{\text{sat}}(P_{\text{base}}, T_{\text{base}}) = \frac{P_{\text{sat}}(T_{\text{base}})}{P_{\text{base}}} = x_{\text{vap}}^{\text{deep}}. \quad (2.19)$$

Below the water condensation zone, the region is subsaturated and hence no condensation occurs. Temperatures below the water condensation zone are obtained by integrating the

dry adiabat ∇_{ad}

$$T(P > P_{\text{base}}) = T_{\text{base}} + \int_{P_{\text{base}}}^P \left(\frac{dT}{dP} \right)_{\text{ad}} dP. \quad (2.20)$$

2.4 Energy Conservation and Thermal Evolution of Model

Conservation of energy implies that the planet's luminosity, given by

$$L_{\text{int}} = 4\pi R^2 \sigma_{\text{SB}} T_{\text{int}}^4 \quad (2.21)$$

must be balanced by the rate of change of its total internal energy. When we have a sequence of progressively cooler models, we calculate the time-step between any two models using the energy conservation equation as in (Fortney et al., 2011)

$$\frac{dL}{dm} = -T \frac{\partial S}{\partial t} \quad (2.22)$$

where dm is the mass of the shell, ∂S is the entropy of the shell, and T is the temperature of the shell. Solving for ∂t , we get the time-step

$$\partial t = -\frac{1}{L} \int_0^M \partial S dm \quad (2.23)$$

3

Results

3.1 Condensation-inhibited Convection

Figure 3.1 shows the results of our initial, exploratory models. We show α with respect to pressure, vapor mole fraction, and temperature. These static models are run for a variety of T_{10} 's, the planet's temperature at $P = 10$ bars. As the bulk water abundance for Uranus and Neptune is unconstrained (Guillot, 1995), the model runs use three different values of q_{deep} , searching for deep water abundances and evolutionary phases for which convection is inhibited by water condensation. In these exploratory models, we only consider the model Uranus. We find that for $q_{\text{deep}} = 0.05$, no condensation-inhibited convection occurs. In other words, α (Eqn. 2.13) never takes on negative values with this concentration of water vapor, hence the condition for stability is never met. However, for larger values of q_{deep} , we find that α does take on negative values (see rows 2 and 3 in Figure 3.1). These finding are in agreement with (Friedson & Gonzales, 2017; Leconte et al., 2017). The shaded regions of the plots indicate the pressure-space over which α is negative. For

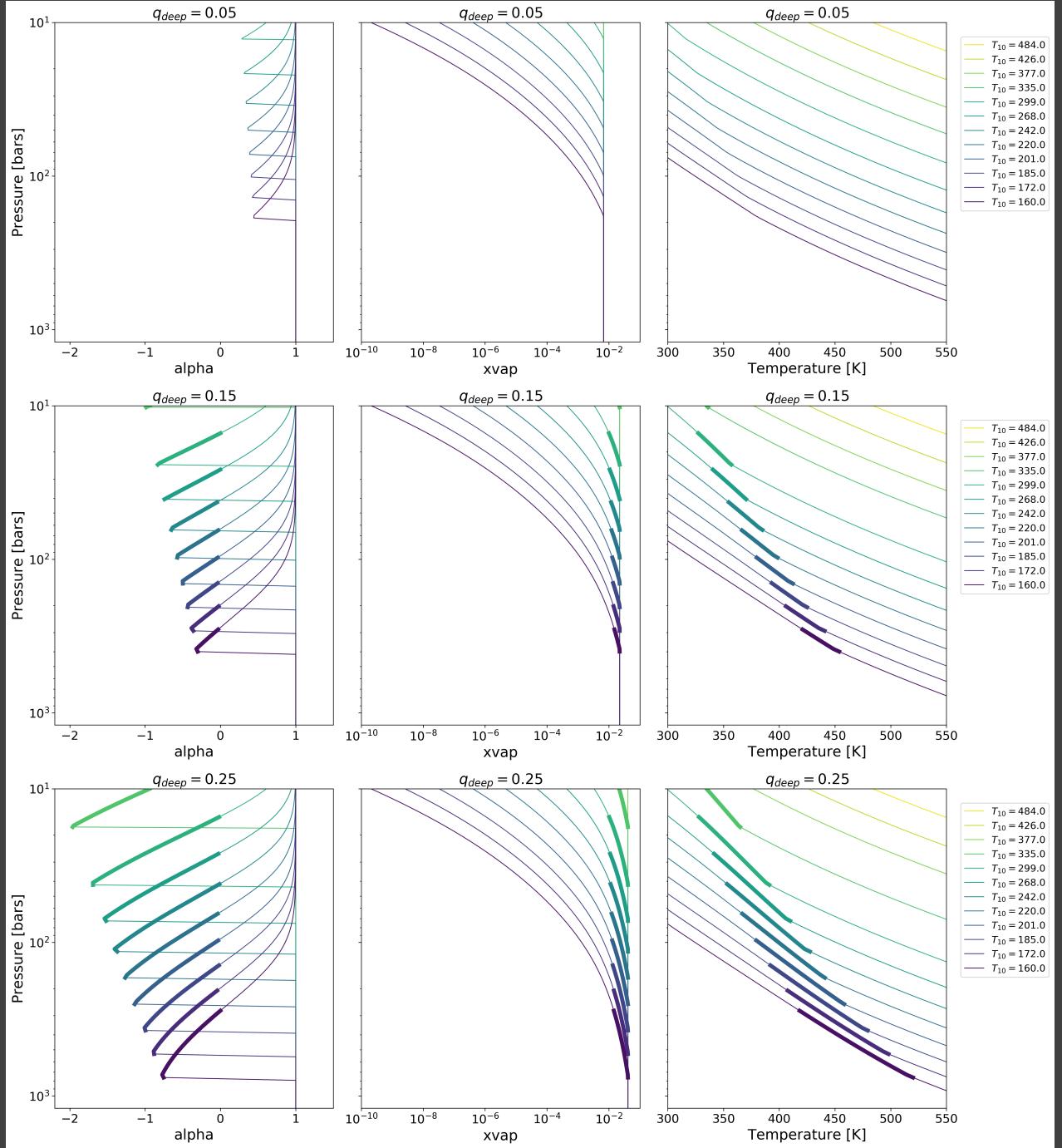


Figure 3.1: Each row represents a different value for q_{deep} . For $q_{deep} = 0.05$, no stable condensation zone forms. For $q_{deep} = 0.15$ and $q_{deep} = 0.25$, convection is inhibited by condensation. The shaded regions show the extent of when α is negative.

which indicates the onset of condensation, at which point the lapse rate has a shallower slope. For the larger values of q_{deep} , where α takes on negative values, we see the onset of condensation-inhibited convection and the establishment of a radiative zone. In the plots, the water condensation zones are represented by the horizontal discontinuities moving from left to right. As the planet cools, these radiative zones descend deeper into the planet's interior. When the radiative zones are established, the interior below the zone becomes much warmer. In Figure 3.2, we highlight the effect of a warming interior. In this figure, we have overlaid the profiles for $q_{\text{deep}} = 0.25$ (exhibiting stable water condensation zones) over the profile for $q_{\text{deep}} = 0.05$ (no stable zones). From this plot, one can see that the presence of a radiative zone creates a temperature jump such that a given T_{10} appears to look like an earlier T_{10} . In other words, we find that the steep temperature increase caused by the presence of a radiative zone causes the interior to be much hotter than one would find using a simple moist adiabatic model with no stable layers. So, for a fixed T_{10} , while sub-critical ($q_{\text{deep}} = 0.05$) and super-critical ($q_{\text{deep}} = 0.25$) models may appear identical near the surface, the super-critical model has a much warmer interior, one that resembles an earlier evolutionary track at a higher T_{10} .

Looking at the adjacent x_{vap} plots, we can see that x_{vap} follows its saturated value. At the bottom of the radiative zone, the vapor mole fraction equals its deep water value, which is the condition that sets the base of the condensation zone.

3.3 Thermal Evolution of Uranus and Neptune

In Figure 3.4, we display the results of evolutionary tracks that consider separately the evolution of a dry adiabat, a moist adiabat with condensation but no stable radiative

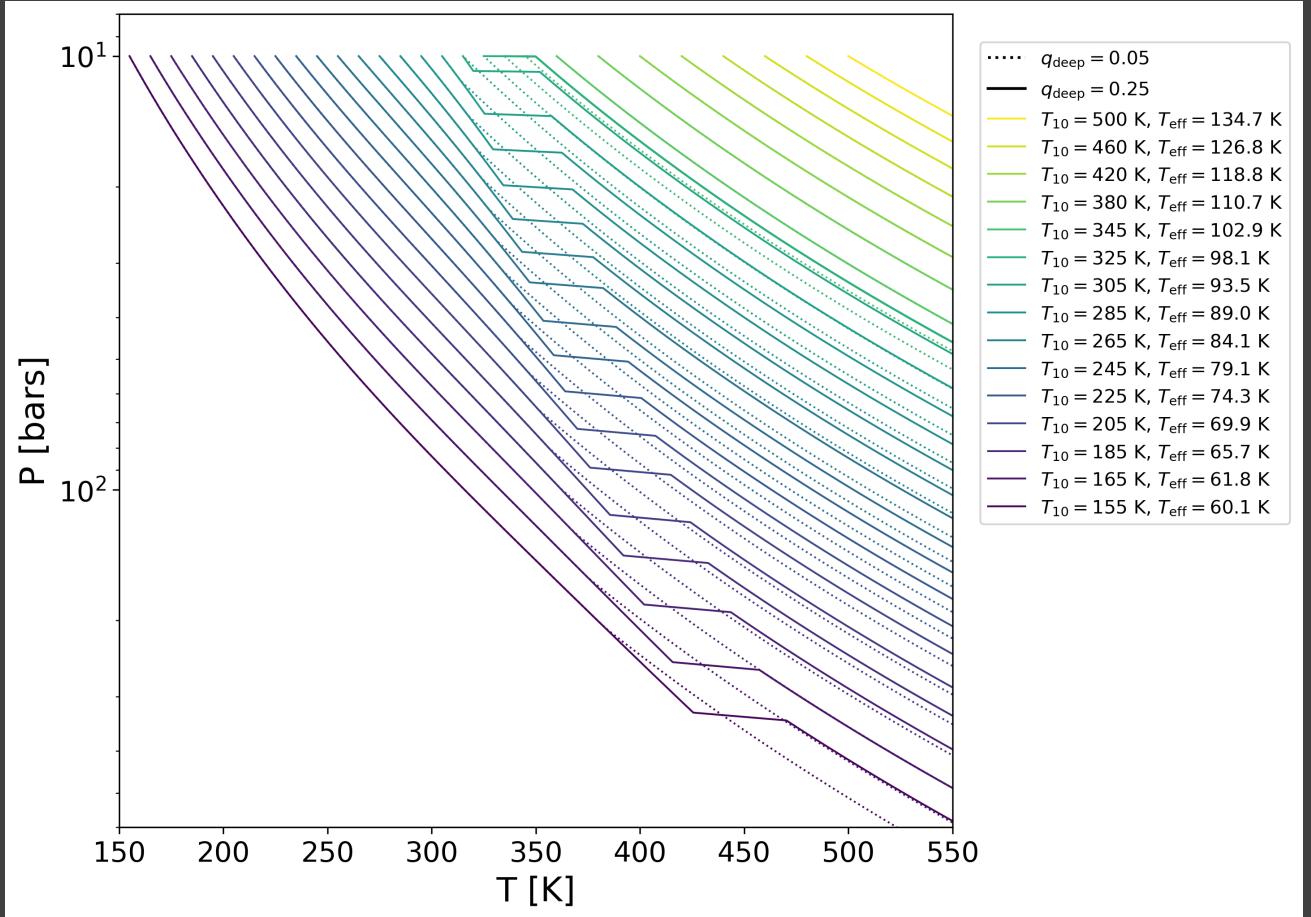


Figure 3.2: The solid lines represent the pressure-temperature profile for $q_{\text{deep}} = 0.25$, and the dashed lines for $q_{\text{deep}} = 0.05$. Looking at recent T_{10} 's, interior temperatures for $q_{\text{deep}} = 0.25$ jump to an earlier T_{10} .

zone, and a moist adiabat with condensation containing stable radiative zones. For all of these evolutionary tracks, we assume $q_{\text{deep}} = 0.25$. Looking at these evolutionary tracks, the coolest scenario at present time, is a moist adiabat that is never stable against convection. The moist adiabat that is stable against convection has the warmest outcome at present time. In Figure 3.5 (Uranus) and Figure 3.6 (Neptune), we consider the impact of different deep water concentrations on the thermal evolution of Uranus and Neptune. As the planets cool, their radiative zones descend deeper into the interior, as we saw in Figure 3.3. This

behavior is also noticeable in the thermal evolution plots. Looking at T_{eff} at 7×10^7 Gyr, the onset of condensation-inhibited convection occurs, resulting in a discontinuous temperature drop. The same behavior is seen in the T_{10} plots for both planets, however, by this time the radiative zone has descended deeper, later in time at around 7×10^8 Gyr. Larger q_{deep} 's result in warmer Uranus and Neptune at present time. We also look at the impact of q_{deep} on the evolution of planetary radius and find that larger values of q_{deep} tend to converge more closely toward the presently observed radius for both Uranus and Neptune in these simulations.

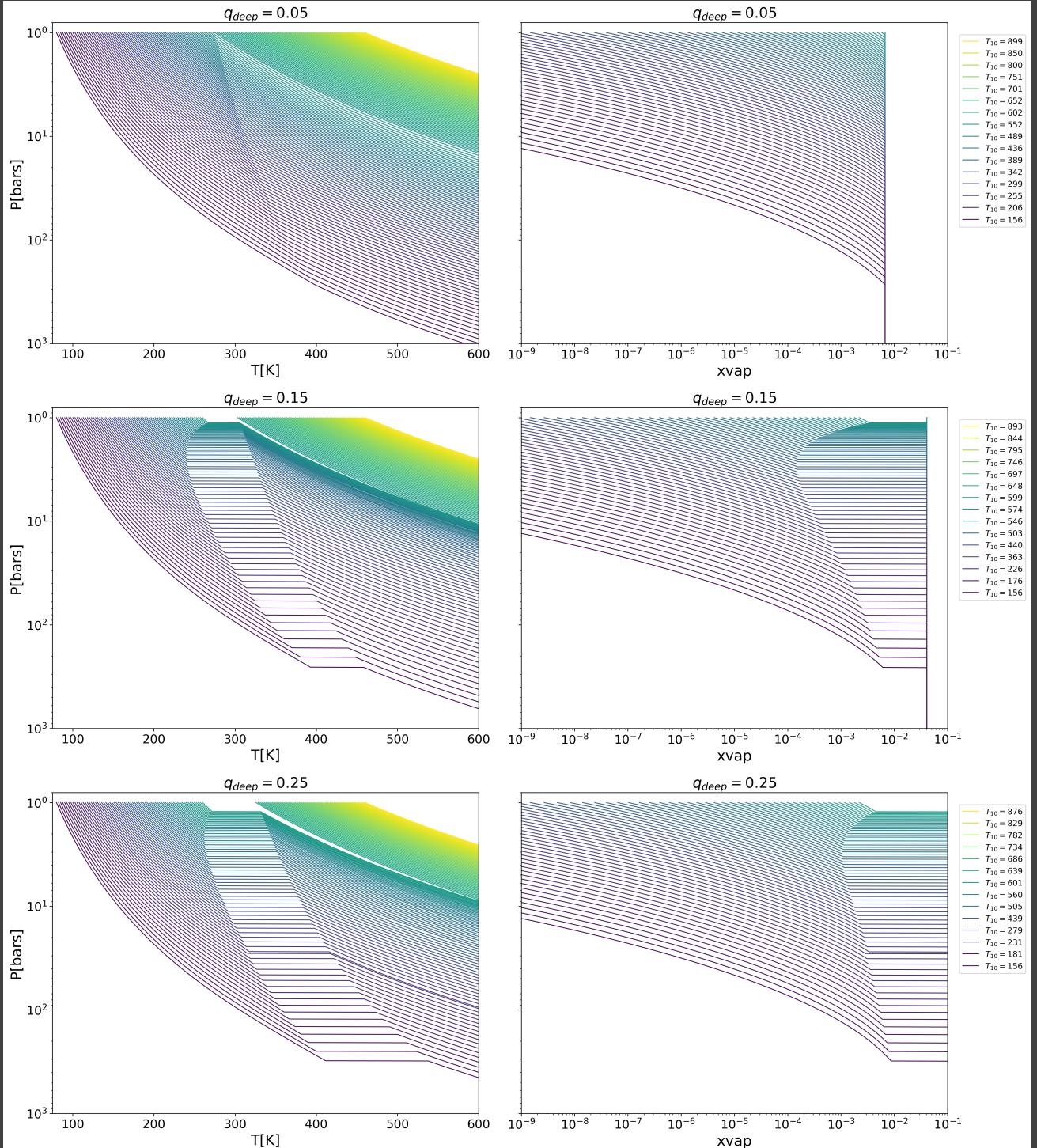


Figure 3.3: These plots were generated using our model Uranus. Again, from top to bottom, we move from $q_{\text{deep}} = 0.05$, 0.15, and 0.25, respectively. T_{10} 's range from hotter (yellow) to cooler (purple), more recent temperatures. For $q_{\text{deep}} = 0.05$, no stable radiative zones are formed. The kink visible in the middle of the top left plot represents the transition from a moist to dry adiabat. Condensation occurs, but no stability is achieved. For $q_{\text{deep}} = 0.15$ and $q_{\text{deep}} = 0.25$, stable radiative zones are formed, as indicated by the discontinuous temperature jumps moving left to right.

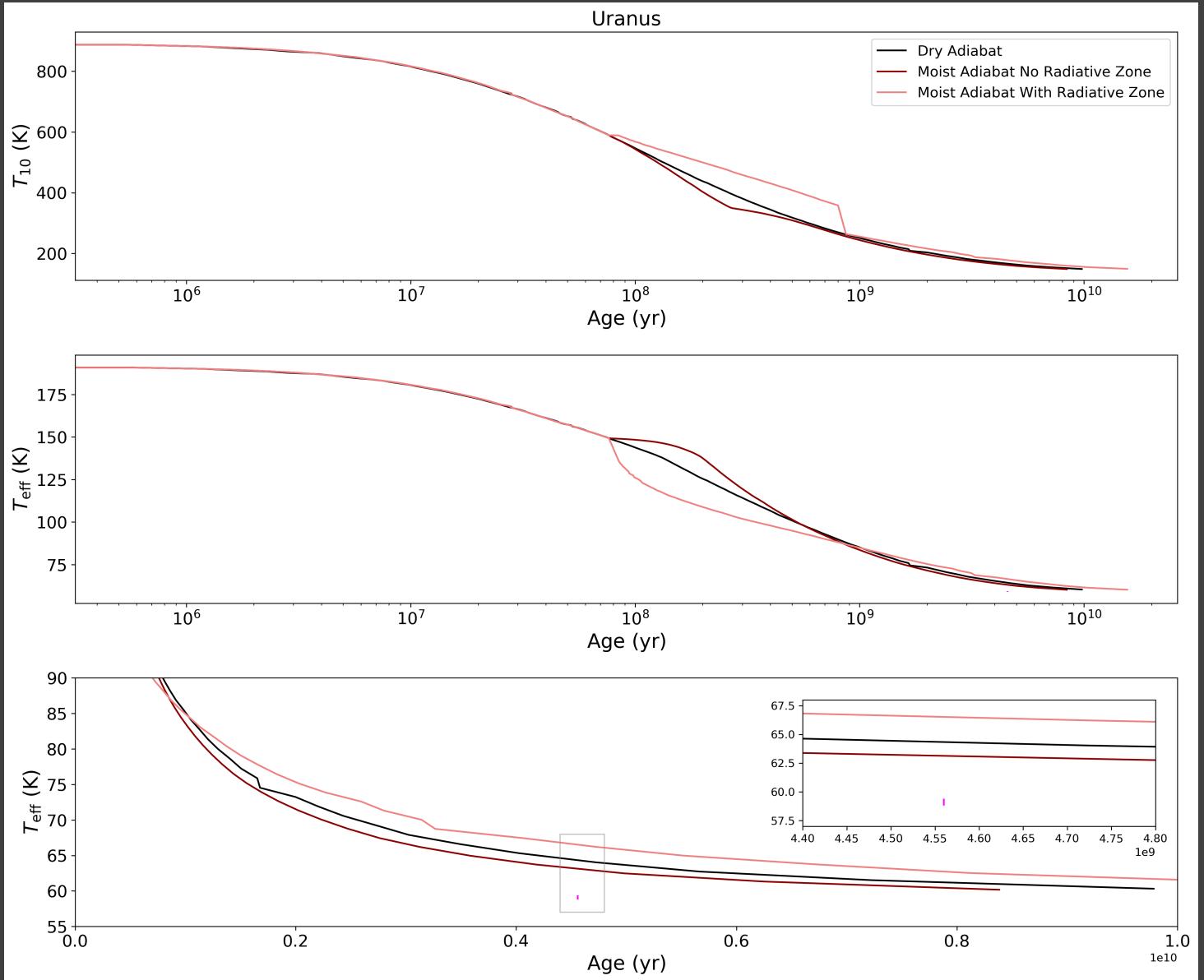


Figure 3.4: The black line represents the thermal evolution for a dry adiabat. The dark red line represents the thermal evolution for a moist adiabat that does not allow for the formation of a stable radiative layer. The light red line represents the thermal evolution of a moist adiabat that does allow for the formation of a stable radiative zone. The fuchsia dot on the lower plot represent the currently observed effective temperature of Uranus with error range.

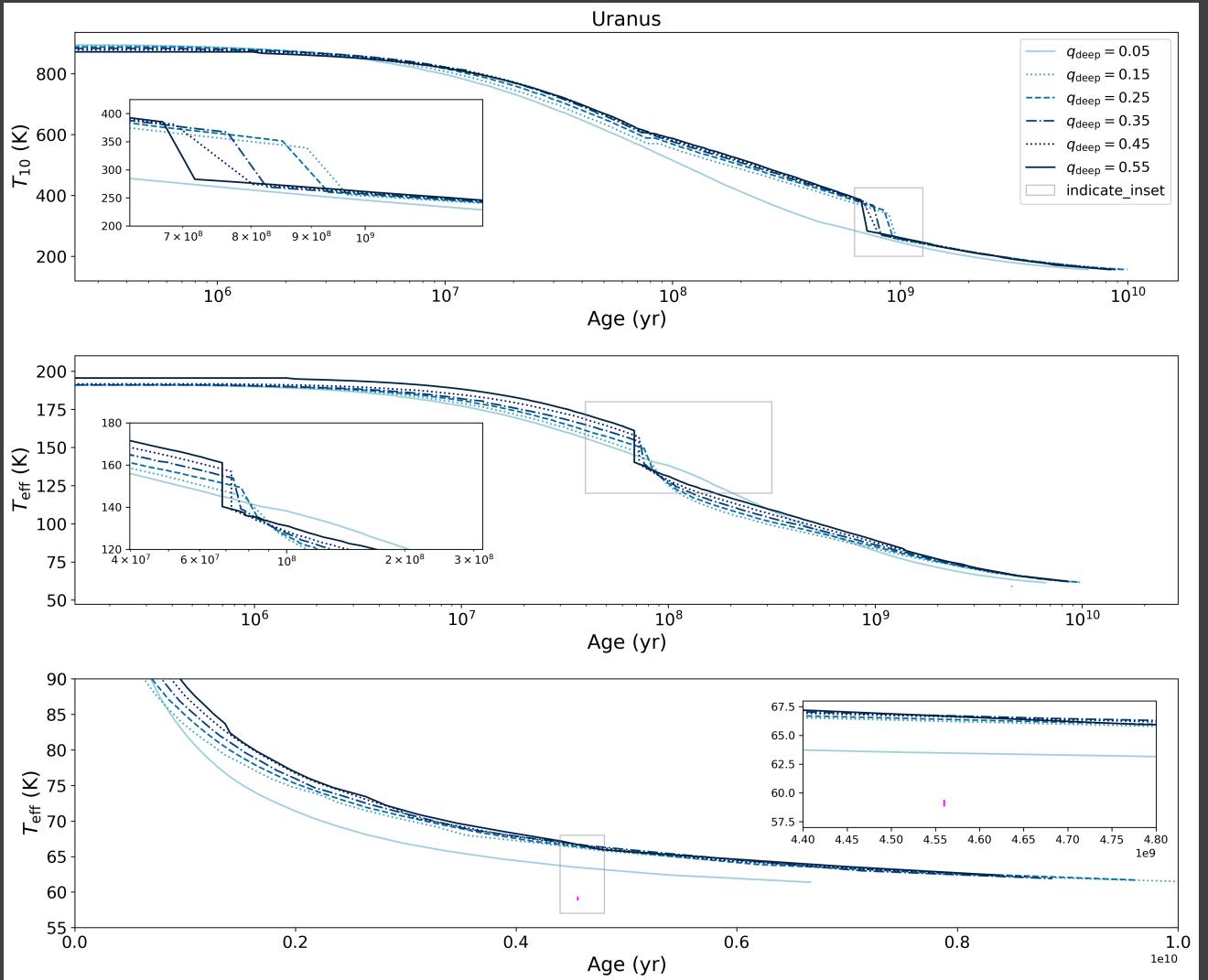


Figure 3.5: The curves in these plots represent thermal evolution tracks for different values of q_{deep} . Dark blue is the largest concentration of water vapor, at $q_{\text{deep}} = 0.55$ and the light blue line is the least concentration of water vapor at $q_{\text{deep}} = 0.05$. For $q_{\text{deep}} = 0.05$, there is no onset of condensation-inhibited convection and no rapid cooling episode. For larger values of q_{deep} there is a rapid cooling episode for T_{eff} at around 7×10^7 Gyr. Similarly, a rapid cooling episode is visible deeper down in the interior as seen in the T_{10} curves at around 8×10^8 Gyr. The insets zoom in on periods of rapid cooling. The vertical fucshia line represents the current T_{eff} with error range.

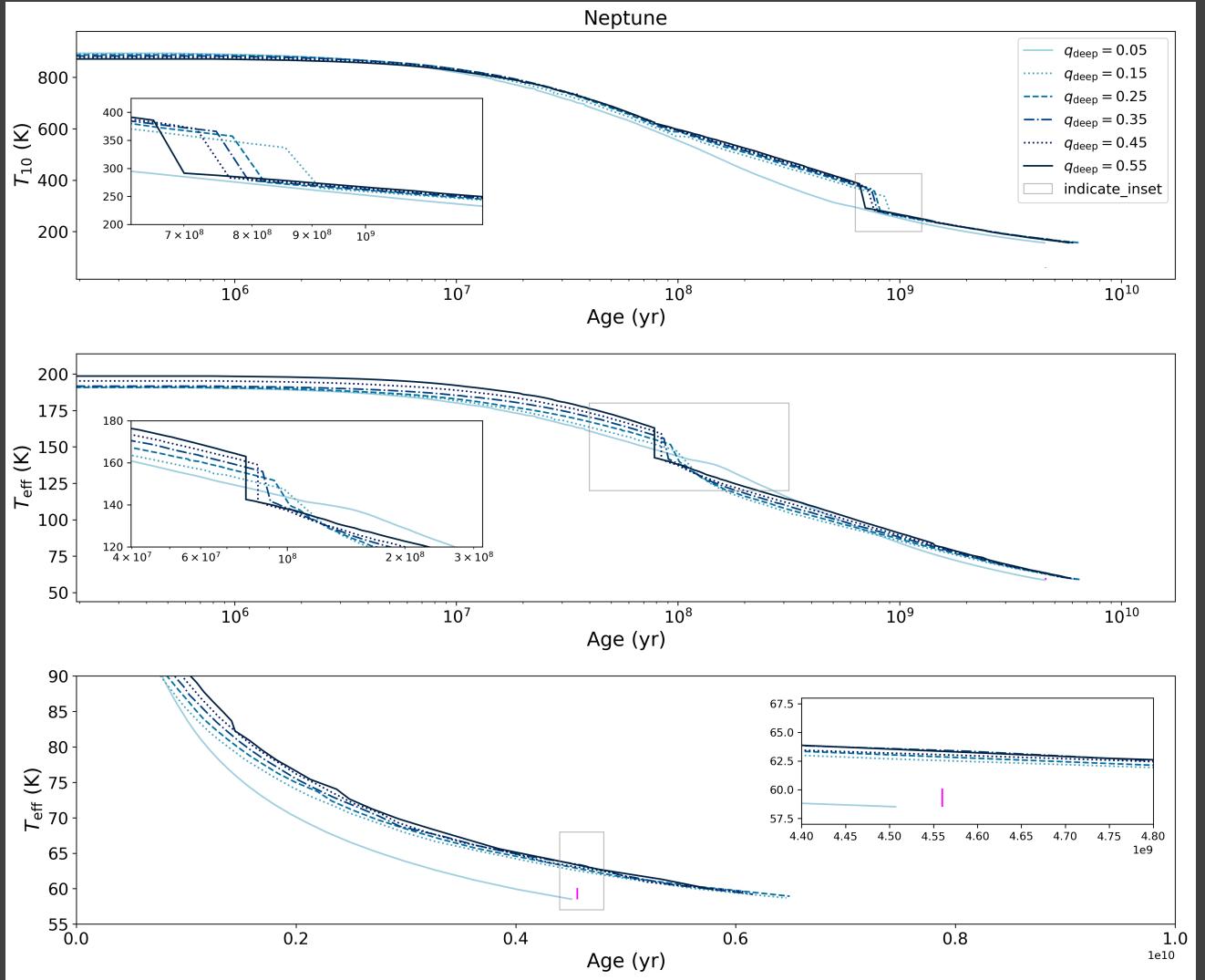


Figure 3.6: Similar to the Uranus plots, The curves in these plots represent thermal evolution tracks for different values of q_{deep} . Dark blue is the largest concentration of water vapor, at $q_{\text{deep}} = 0.55$ and the light blue line is the least concentration of water vapor at $q_{\text{deep}} = 0.05$. For $q_{\text{deep}} = 0.05$, there is no onset of condensation-inhibited convection and no rapid cooling episode. For larger values of q_{deep} there is a rapid cooling episode for T_{eff} at around 7×10^7 Gyr. Similarly, a rapid cooling episode is visible deeper down in the interior as seen in the T_{10} curves at around 8×10^8 Gyr. The insets zoom in on periods of rapid cooling. The vertical fucshia line represents the current T_{eff} with error range.

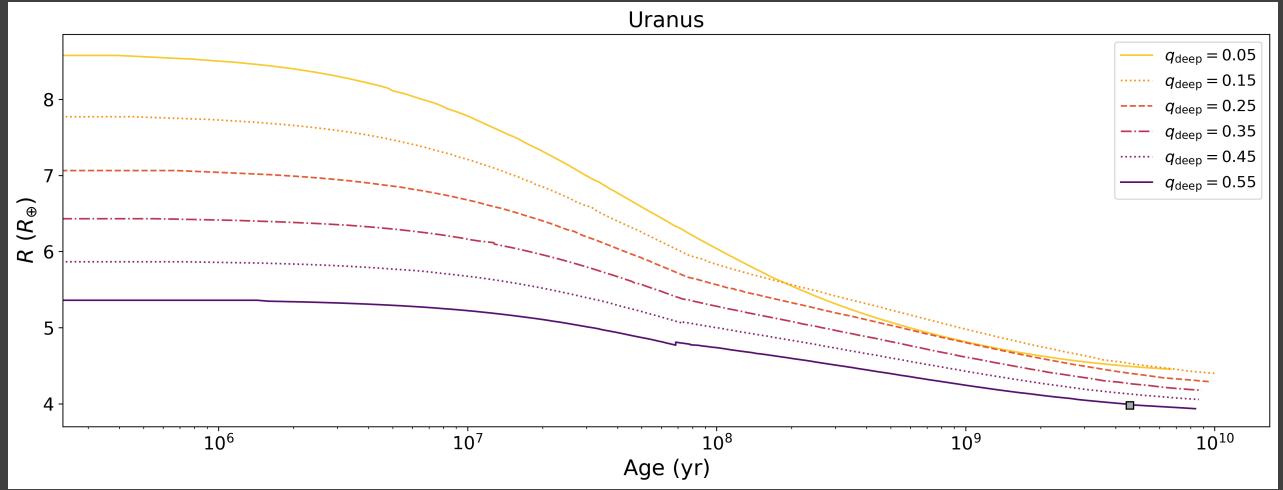


Figure 3.7: This thermal evolution plot shows the impact of different deep water concentration on the radius on model Uranus as it cools. The gray square represents the current observed radius.

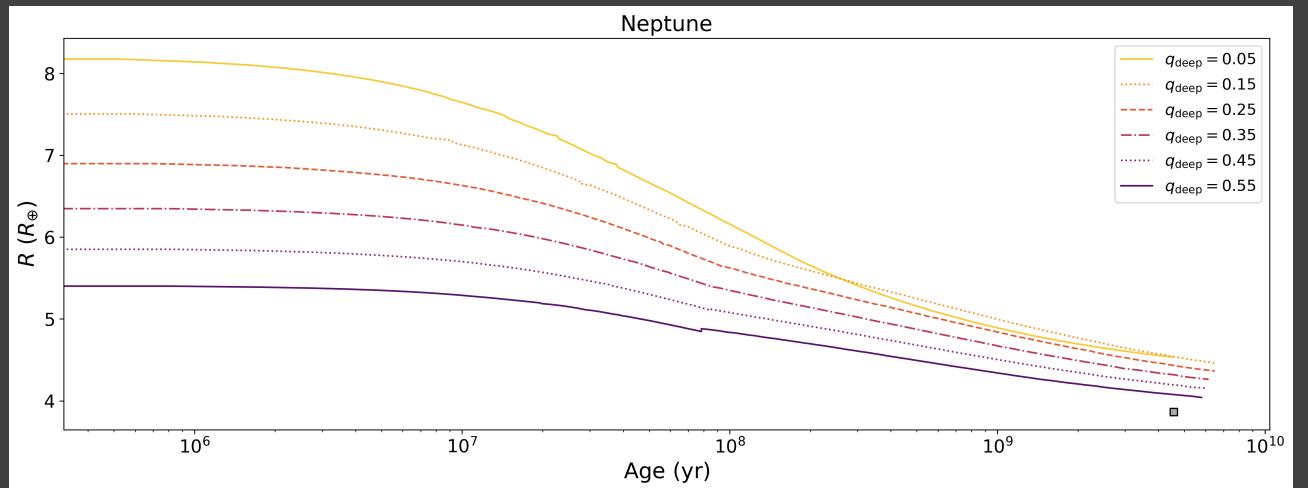


Figure 3.8: This thermal evolution plot shows the impact of different deep water concentration on the radius on model Neptune as it cools. The gray square represents the current observed radius.

4

Discussion and Conclusions

We set out to investigate the impact of water condensation zones on the thermal evolution of our solar system ice giants. It has been speculated that such thermal boundary layers could act as an imperfect insulator, trapping heat below and allowing the envelope above the boundary layer to cool more rapidly (Nettelmann et al., 2016)(Friedson & Gonzales, 2017)(Leconte et al., 2017)(M. Podolak, 1991)(L. Scheibe, 2019). It seems plausible that such interiors could explain the problem with Uranus appearing to have no intrinsic temperature. And, while our analysis suggests that moist-adiabatic interiors have a significant impact on the heat flow and thermal evolution of ice giants, making a case for the inclusion of moist adiabats in contemporary interior structure models, our findings are nonetheless inconclusive on the problem of Uranus. We do find that incorporating a moist adiabat into our interior structure model does result in a cooler model Uranus and Neptune than would otherwise be seen with a purely dry model. However, when we add stable radiative zones to the interior, we find in the planet's past a period of rapid cooling that results in a cooler effective temperature at around 7×10^7 Gyr. However, both model

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