

Notes on rigid body dynamics

Plücker coordinates

Motion

$$\underline{m}_o = \begin{Bmatrix} M_x \\ M_y \\ M_z \\ M_{ox} \\ M_{oy} \\ M_{oz} \end{Bmatrix} = \begin{Bmatrix} M \\ M_o \end{Bmatrix}$$

rotation
translation

These are Plücker coordinates for $m \in M^6$

Force

$$\underline{f}_o = \begin{Bmatrix} f_{ox} \\ f_{oy} \\ f_{oz} \\ f_x \\ f_y \\ f_z \end{Bmatrix} = \begin{Bmatrix} f_o \\ f \end{Bmatrix}$$

moment about 0
force

These are Plücker coordinates for $f \in F^6$

$$\text{Inner product: } m \cdot f = \underline{m}_o^T \underline{f}_o$$

Coordinate transformation

Denote ${}^B X_A$ as transform from A to B coordinates (for motion vector)

and ${}^B X_A^*$ for force vector

For rotation, we use a 3×3 rotation matrix E that transforms 3d vectors from A to B coordinates. Then

$${}^B X_A = \begin{bmatrix} E & 0 \\ 0 & E \end{bmatrix}$$

$$\begin{aligned} \text{e.g. } v &= v_x^A e_x^A + v_y^A e_y^A + v_z^A e_z^A \\ &= v_x^B e_x^B + v_y^B e_y^B + v_z^B e_z^B \\ v_x^B &= (e_x^B \cdot e_x^A) v_x^A + (e_x^B \cdot e_y^A) v_y^A + (e_x^B \cdot e_z^A) v_z^A \end{aligned}$$

$$E = \begin{bmatrix} {}^B e_x^A & {}^B e_y^A & {}^B e_z^A \\ {}^B e_x^A & {}^B e_y^A & {}^B e_z^A \\ {}^B e_x^A & {}^B e_y^A & {}^B e_z^A \end{bmatrix}$$

For translation by r (from 0 to P)

$$\begin{aligned} m &= m \\ m_p &= m_0 + m \times r \end{aligned} \quad \left. \right\}$$

$${}^B X_A = \begin{bmatrix} 1 & 0 \\ -r^x & 1 \end{bmatrix} \quad r^x_a \equiv r \times a, \quad (r^x)^T = -r^x$$

or for force vectors, since

$$\begin{aligned} f_r &= f_o - r \times f \\ f &= f_o \end{aligned} \rightarrow {}^B X_A^* = \begin{bmatrix} 1 & -r^x \\ 0 & 1 \end{bmatrix}$$

since moment term is F_o

In general,

$${}^B X_A = \begin{bmatrix} E & 0 \\ 0 & E \end{bmatrix} \begin{bmatrix} 1 & 0 \\ -r^x & 1 \end{bmatrix}$$

translate from center of A to center of B,
then rotate

$${}^B X_A^* = \begin{bmatrix} E & 0 \\ 0 & E \end{bmatrix} \begin{bmatrix} 1 & -r^x \\ 0 & 1 \end{bmatrix}$$

The inverses are

$${}^A X_B = \begin{bmatrix} 1 & 0 \\ r_x & 1 \end{bmatrix} \begin{bmatrix} E^T & 0 \\ 0 & E^T \end{bmatrix} = ({}^B X_A)^T \quad {}^A X_B^* = \begin{bmatrix} 1 & r_x \\ 0 & 1 \end{bmatrix} \begin{bmatrix} E^T & 0 \\ 0 & E^T \end{bmatrix} = ({}^B X_A)^T$$

Spatial cross products

$$\begin{aligned} \dot{m} &= \hat{v} \times m & \left. \begin{aligned} &\text{vectors moving with velocity } v \text{ (i.e. fixed to} \\ &\text{a rigid body w.r.t. that velocity)} \end{aligned} \right\} \\ \dot{f} &= \hat{f} \times^* f \end{aligned}$$

Then, $\hat{v}_o^x = \begin{bmatrix} \omega \\ v_o \end{bmatrix}^x = \begin{bmatrix} \omega^x & 0 \\ v_o^x & \omega^x \end{bmatrix}$ $\hat{v}_o^{xx} = \begin{bmatrix} \omega \\ v_o \end{bmatrix}^x = \begin{bmatrix} \omega^x & v_o^x \\ 0 & \omega^x \end{bmatrix} = -(\hat{v}_o^x)^T$

So, for example

$$\begin{aligned} \hat{v}_o \times \hat{m} &= \begin{bmatrix} \omega^x & 0 \\ v_o^x & \omega^x \end{bmatrix} \begin{bmatrix} m \end{bmatrix} = \begin{bmatrix} \omega \times m \\ v_o \times m + \omega \times m_o \end{bmatrix} \\ \hat{v}_o \times^* \hat{f} &= \begin{bmatrix} \omega^x & v_o^x \\ 0 & \omega^x \end{bmatrix} \begin{bmatrix} f_o \end{bmatrix} = \begin{bmatrix} \omega \times f_o + v_o \times f \\ \omega \times f \end{bmatrix} \end{aligned} \quad \left. \begin{aligned} &\text{Note that} \\ &{}^B X_A^* ({}^A v \times^* {}^B X_A) = ({}^B X_A {}^A v) \times^* \\ &\text{Thus, } {}^B X_A^* ({}^A v \times^* {}^A f) = {}^B v \times^* {}^B f \end{aligned} \right\}$$

Differentiation in Moving Coordinates

If A is coordinate system associated with a frame moving with velocity v_A ,

$$\begin{aligned} {}^A \left(\frac{dm}{dt} \right) &= \frac{d {}^A m}{dt} + {}^A v_A \times {}^A m \\ &\text{Componentwise derivatives of } {}^A m \quad \text{rate of change due to frame motion} \end{aligned}$$

Similarly,

$${}^A \left(\frac{df}{dt} \right) = \frac{d {}^A f}{dt} + {}^A r_A \times {}^A f$$

This leads to the expression for rate of change of ${}^B X_A$:

$$\frac{d}{dt} {}^B X_A = {}^B (v_A - v_B) \times {}^B X_A$$

Acceleration

Define the spatial acceleration

$$\hat{a}_o = \frac{d}{dt} \left\{ \begin{bmatrix} \omega \\ v_o \end{bmatrix} \right\} = \left\{ \begin{bmatrix} \dot{\omega} \\ \dot{v}_o \end{bmatrix} \right\}$$

Keep in mind that \dot{v}_o is not velocity of one particular body-fixed point, but a measure of flow of points through O (i.e. it is Eulerian, not Lagrangian)

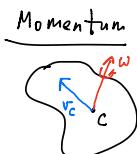
Thus \dot{v}_o is not acceleration of body-fixed point but a measure of rate of change of flow. (It is not the material derivative, but the Eulerian derivative)

If $r(t)$ is position of a fixed point O' on body, and $O = O'$ at t , then

$$(\ddot{r}, \dot{\omega}) \rightarrow \text{classical rigid body acceleration} \quad \boxed{\ddot{a}_o' = \begin{Bmatrix} \dot{\omega} \\ \ddot{r} \end{Bmatrix}}$$

$$\text{But } \dot{v}_o = \frac{v_o(t+\delta t) - v_o(t)}{\delta t} = \frac{[\dot{r}(t+\delta t) + \omega \times (-\delta t \dot{r}(t))] - \dot{r}(t)}{\delta t} \rightarrow \dot{r} - \omega \times \dot{r}$$

$$\boxed{\hat{a}_o = \begin{Bmatrix} \dot{\omega} \\ \dot{r} - \omega \times \dot{r} \end{Bmatrix}} \leftarrow \text{a true vector!} \quad (\text{e.g. } \hat{a}_{\text{rel}} = \hat{a}_2 - \hat{a}_1, \text{ for bodies 1,2})$$



Center of mass coincides with C at one instant

mass m rotational inertia \bar{I}_c (about center of mass)

$$\text{Momentum} \begin{cases} \text{linear} & h = m v_c \\ \text{angular} & h_c = \bar{I}_c \omega \end{cases} \quad h_o = h_c + \bar{oC} \times h$$

$$\text{Spatial momentum} \quad \hat{h}_c = \begin{Bmatrix} h_c \\ h \end{Bmatrix} = \begin{Bmatrix} \bar{I}_c \omega \\ m v_c \end{Bmatrix} \quad \hat{h}_o = \begin{Bmatrix} h_o \\ h \end{Bmatrix} = \begin{bmatrix} 1 & \bar{oC} \times \\ 0 & 1 \end{bmatrix} \hat{h}_c$$

$$(\text{In general, } {}^B \hat{h} = {}^A X_A^{-1} \hat{h})$$

$$\text{kinetic energy} \quad \frac{1}{2} \hat{h} \cdot \hat{h}$$

Inertia

$$I: M^6 \leftrightarrow F^6 \quad (\text{from velocity to momentum})$$

$$\hookrightarrow h = I v$$

$$\text{In particular, } h_c = \underbrace{\begin{bmatrix} \bar{I}_c & 0 \\ 0 & m \end{bmatrix}}_{I_c} v_c \quad c = \bar{oC}$$

Then,

$$h_o = \begin{bmatrix} 1 & c \times \\ 0 & 1 \end{bmatrix} I_c v_c = \begin{bmatrix} 1 & c \times \\ 0 & 1 \end{bmatrix} I_c \begin{bmatrix} 1 & 0 \\ (c \times)^T & 1 \end{bmatrix} v_o \rightarrow I_o = \begin{bmatrix} \bar{I}_c + m c \times c \times^T & m c \times \\ m c \times^T & m \end{bmatrix}$$

We can write \underline{I} as a symmetric dyadic tensor

$$\underline{I} = \sum_{i=1}^6 g_i g_i \cdot \quad g_i \in F^6$$

Note that $\dot{g}_i = v \times^* g_i$ (change only due to body motion)

Thus,

$$\boxed{\frac{d\underline{I}}{dt} = v \times^* \underline{I} - \underline{I} v \times}$$

Can also show that $\boxed{^B \underline{I} = {}^B X_A^* \underline{I} {}^A X_B}$

Kinetic energy $T = \frac{1}{2} v \cdot \underline{I} v$

Equation of motion

$$\begin{aligned} f &= \frac{d}{dt} (\underline{I} v) = \underline{I} a + (v \times^* \underline{I} - \underline{I} v \times) v \\ &= \underline{I} a + \underbrace{v \times^* \underline{I} v}_{P, \text{ bias force}} \quad \leftarrow \begin{array}{l} P \text{ could also include other} \\ \text{known forces (gravity, etc)} \end{array} \end{aligned}$$

Inverse inertia

$\underline{\Phi} = \underline{I}^{-1} \rightarrow$ range of $\underline{\Phi}$ is subspace of M^6 in which the body is free to move

(rank of $\underline{\Phi}$ = no. degrees of motion freedom)

A note on reference frames

Technically, each tensor and vector, when expressed in any body fixed coordinates, e.g. body system or center-of-mass system, can be regarded as transformed to a system that is fixed, but instantaneously coincides with the body-fixed system.

e.g. frames b and $b_i \rightarrow I_{bi} = {}^{bi}X_b^* I_b {}^bX_b$, is inertia in body

moving with body

fixed but instantaneously coincident with

system

$$\frac{d}{dt} {}^bX_{bi} = {}^b(v_{bi} - v_b) \times {}^bX_{bi} = -v_b \times {}^bX_{bi}$$

$$\frac{d}{dt} I_{bi} = \frac{d}{dt} {}^bX_b^* I_b {}^bX_{bi} + {}^bX_b^* I_b \frac{d}{dt} {}^bX_{bi}$$

$$= {}^bX_b^* v_b \times {}^bI_b {}^bX_{bi} - {}^bX_b^* I_b v_b \times {}^bX_{bi}$$

Thus,

$$\begin{aligned} \frac{dh_{bi}}{dt} &= \frac{d}{dt} (I_{bi} v_{bi}) = {}^bX_b^* v_b \times {}^bI_b v_b - {}^bX_b^* I_b v_b \times v_b + {}^bX_b^* I_b {}^bX_{bi} \alpha_b \\ &= {}^bX_b^* (v_b \times {}^bI_b + I_b \alpha_b) = v_{bi} \times {}^b h_{bi} + I_{bi} \alpha_{bi} \end{aligned}$$

$$\boxed{\frac{dh_{bi}}{dt} = v_{bi} \times {}^b h_{bi} + I_{bi} \alpha_{bi}}$$

So ${}^bX_{bi}$ is instantaneously the identity, but has non-zero rate of change.

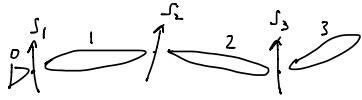
Also, in a joint constraint, $v_j = {}^jX_{j-1} v_{j-1} + v_{sj}$, we differentiate to find acceleration with the implicit understanding that these are expressed in j_i coordinates.

$$\text{Then } \frac{d}{dt} ({}^jX_j v_{sj}) = \frac{d}{dt} {}^{ji}X_j v_{sj} + {}^{ji}X_j \alpha_{sj} = {}^{ji}X_j \alpha_{sj} + \underbrace{{}^{ji}v_j \times {}^{ji}v_{sj}}_{c_j}$$

Also, by definition, for two systems j_i and k_i :

$$\boxed{\frac{d}{dt} {}^{ki}X_{ji} = 0}$$

Linked rigid body motion



Velocity across joint is "to" velocity relative to "from" velocity

$$v_{J_i} = v_i - v_{i-1}$$

Assume each joint allows only single degree of freedom

$$v_{J_i} = s_i \dot{q}_i$$

axis vector

$$\text{Thus } v_i = v_{i-1} + s_i \dot{q}_i$$

$$= v_0 + \sum_{j=1}^i s_j \dot{q}_j = v_0 + J_i \dot{q}$$

$$J_i = [s_1 \ s_2 \ \dots \ s_i \ 0 \ 0 \ \dots \ 0]$$

$$\dot{q} = [\dot{q}_1 \ \dots \ \dot{q}_N]^T$$

Acceleration

$$a_{J_i} = a_i - a_{i-1}$$

$$\text{and } a_{J_i} = s_i \ddot{q}_i + \underbrace{s_i \dot{q}_i^2}_{\dot{s}_i = v_i \times s_i}$$

$$\longrightarrow a_i = a_{i-1} + s_i \ddot{q}_i + v_i \times s_i \dot{q}_i$$

$$= a_0 + \sum_{j=1}^i (s_j \ddot{q}_j + v_j \times s_j \dot{q}_j)$$

$$= a_0 + \sum_{j=1}^i s_j \ddot{q}_j + \sum_{j=1}^i \sum_{k=1}^{j-1} s_k \times s_j \dot{q}_j \dot{q}_k$$

$$\text{So we can write } a_i = a_0 + J_i \ddot{q} + J_i \dot{q}$$

$$+ v_0 \times J_i \dot{q}$$

Equations of Motion

General system

$$H(\dot{q})\ddot{q} + C(q, \dot{q}) = \tau$$

generalized inertia generalized acceleration
 bias force generalized velocity
 (value of τ generalized forces
 that produces zero accel.)
 [Coriolis, centrifugal, gravity, retr.]

Kinetic energy $T = \frac{1}{2} \dot{q}^T H \dot{q}$

Note that $\dot{q}, \ddot{q} \in M^n$ $H: M^n \mapsto F^n$

$\tau, C \in F^n$ (q is not a vector, but a point in configuration space)

Generalized motion, force vectors share many properties with spatial vectors (except cross products)

Even more general (without assuming \dot{q} is integral of velocity)

$$H(\dot{q})\ddot{\alpha} + C(q, \dot{q}, \alpha) = \tau$$

$\dot{q} = Q(q)\alpha$ alternative velocity vector

Motion constraints

implicit constraints: $\phi(q) = 0 \xrightarrow{\frac{d}{dt}} K\dot{q} = 0 \xrightarrow{\frac{d}{dt}} K\ddot{q} = \underline{k}$

$$\frac{\partial \phi}{\partial q}$$

$$-K\dot{q}$$

The constrained system has the form

$$H\ddot{q} + C = \tau + \tau_c$$

From Jourdain's principle of virtual power, τ_c delivers zero power along every free velocity direction:

Let $\tau_c = K^T \lambda$. $\tau_c \cdot \dot{q} = 0$ for any \dot{q} allowed by constraint.

$$\rightarrow \tau_c \cdot \ddot{q} = K^T \lambda \cdot \ddot{q} = \lambda^T K \ddot{q} = 0 \quad \lambda \text{ are Lagrange multipliers}$$

Apply constraints:

$$\underbrace{\begin{bmatrix} H & K^T \\ K & 0 \end{bmatrix}}_{n+n_c} \underbrace{\begin{Bmatrix} \ddot{q} \\ \lambda \end{Bmatrix}}_{n+n_c} = \begin{Bmatrix} \tau - C \\ K \end{Bmatrix}$$

$n = \dim(\ddot{q})$
 $n_c = \dim(\lambda)$ (no. of constraints)
 $n_{ic} = \text{rank}(K)$ (no. of indep constraints)
 $(n_{ic} \leq n_c)$

If $n_{ic} = n_c$, system is nonsingular

If $n_{ic} < n_c$, system is overconstrained (?)

Vector Subspaces

If system is constrained to move only in subspace $S \subset M^n$, then

$$\dot{q} \in S = \text{null}(K)$$

If $V = S_1 \oplus S_2$ where V is a vector space and S_1, S_2 are subspaces
 $(S_1 \cap S_2 = \{0\}, \dim(S_1) + \dim(S_2) = \dim(V))$
then any $v \in V$ can be written

$$v = \underbrace{s_1}_{\alpha_1} \alpha_1 + \underbrace{s_2}_{\alpha_2} \alpha_2 = \begin{bmatrix} s_1 & s_2 \end{bmatrix} \begin{Bmatrix} \alpha_1 \\ \alpha_2 \end{Bmatrix}$$

s_1 columns are basis of S_1
 s_2 " " " " " S_2

Note that $[s_1, s_2]$ is basis for V , so

$$\begin{Bmatrix} \alpha_1 \\ \alpha_2 \end{Bmatrix} = [s_1, s_2]^{-1} v$$

Suppose U and V are vector spaces, where $U = V^*$ (the dual of V)
 Then, for $u \in U$ and $v \in V$, if $u \cdot v = 0$, they are orthogonal

If $Y_1 \subseteq U$, $Y_2 \subseteq V$, and every element of Y_1 is orthogonal to every element in Y_2 , then $Y_1 \perp Y_2$.

The set of all $v \in V$ for which $v \cdot u = 0$ for all $u \in Y \subseteq U$, is the orthogonal complement of Y .

If \dot{q} is restricted to $S \subseteq M^n$, then $\tau_c \in S^\perp \subseteq F^n$

Example:

Given motion subspace $S \subseteq M^6$, and spatial inertia $I : M^6 \rightarrow F^6$
 then can define two subspaces $T_a = IS$ and $T_c = S^\perp$, with property

$$F^6 = T_a \oplus T_c$$

Thus, can decompose any f into $f = f_a + f_c$.

To find f_a and f_c from f

$$S^T f = S^T f_a + \cancel{S^T f_c} = \underbrace{S^T f_a}_{IS\alpha} = S^T I S \alpha \implies \alpha = (S^T I S)^{-1} S^T f$$

$\therefore \boxed{f_a = IS(S^T I S)^{-1} S^T f}$

and $f_c = f - f_a$.

apparent inverse inertia

$f_a \rightarrow$ part that creates acceleration of body
 $f_c \rightarrow$ part opposed by constraint force ($-f_c$)