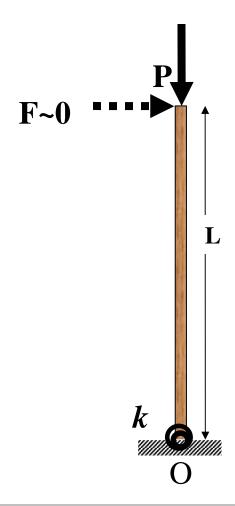
## Stability of a column



We first consider a column hinged at the base and loaded with a vertical force P. It is obvious that a slight disturbance in the form of a small horizontal force F can disturb the equilibrium. We consider the case where the hinge has a torsional stiffness k and analyze the stability of this equilibrium.

Will the column come back to its original position in case of a disturbance or not?

## Stability of a column



If the disturbance is minimal we are justified in considering F~0. Note that the deviation shown in figure is highly exaggerated for clarity.

$$\sum M_{o} = 0$$

$$\Rightarrow PL\sin\theta + FL\cos\theta = k\theta$$

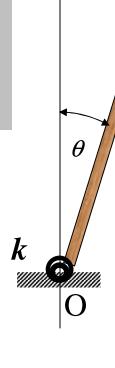
$$F = 0, \theta \to 0$$

$$\Rightarrow PL\theta + (0)L = k\theta$$

$$\Rightarrow P = \frac{k}{L}$$

$$P > \frac{k}{L} \Rightarrow Stable$$

$$P < \frac{k}{L} \Rightarrow Unstable$$



## Stability of a column

We now consider a small disturbance  $\delta\theta$  in order to study the nature of equilibrium

$$\sum M_O = 0$$

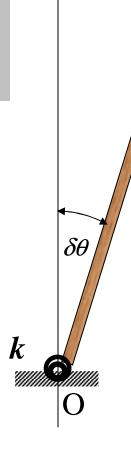
$$\Rightarrow PL\sin\delta\theta - k\delta\theta = 0$$

$$\Rightarrow PL\delta\theta - k\delta\theta = 0$$

$$\Rightarrow (PL - k) \delta\theta = 0$$

Two possibilities exist

$$PL - k = 0, \delta\theta = 0$$



$$\delta\theta = 0$$

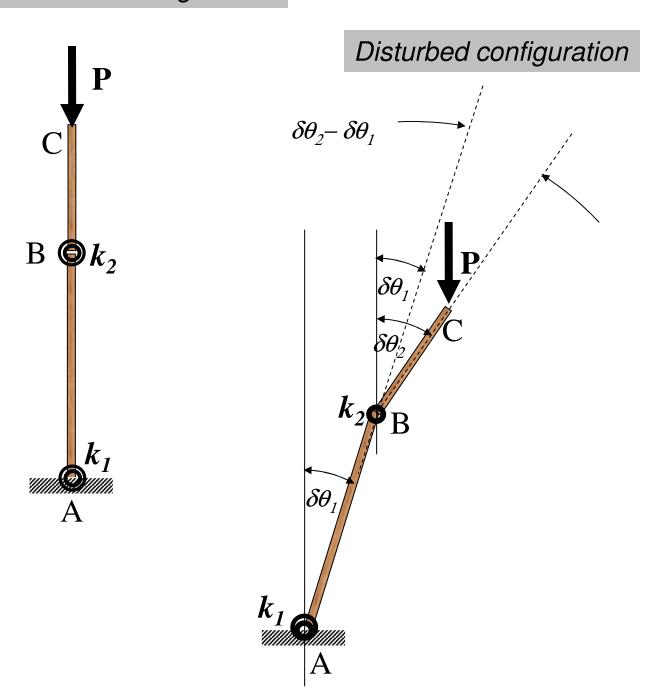
Stable shape – beam is vertical

$$P = P_{cr} = \frac{k}{L}$$

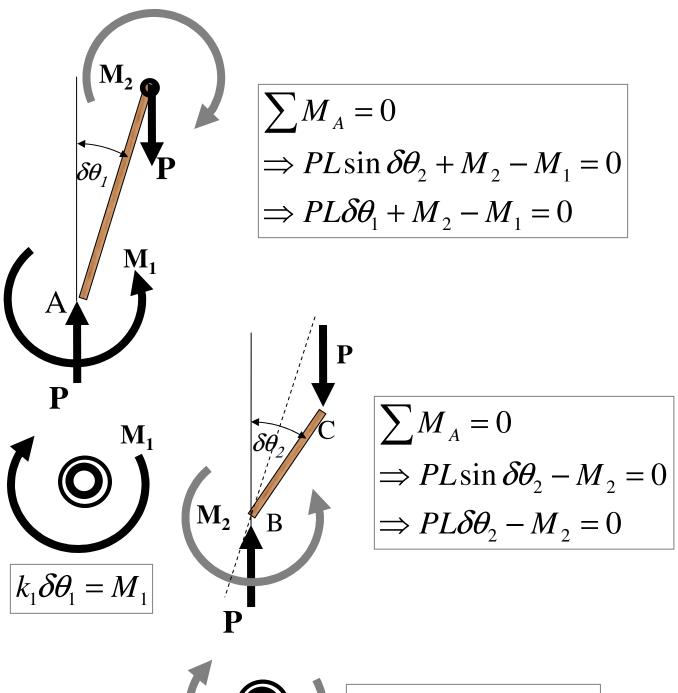
Maximum load for which equilibrium is always restored

We consider a 2 DOF system to elaborate and understand this idea

## Undisturbed configuration



# Free body analysis of individual segments





$$k_2(\delta\theta_2 - \delta\theta_1) = M_2$$

#### Equilibrium equations

$$PL\delta\theta_{2} - M_{2} = 0$$

$$\Rightarrow PL\delta\theta_{2} - k_{2} (\delta\theta_{2} - \delta\theta_{1}) = 0$$

$$PL\delta\theta_{1} + M_{2} - M_{1}$$

$$\Rightarrow PL\delta\theta_{1} + k_{2} (\delta\theta_{2} - \delta\theta_{1}) - k_{1}\delta\theta_{1} = 0$$

#### Rearranging in matrix form

$$\begin{bmatrix} k_2 & PL - k_2 \\ PL - k_1 - k_2 & k_2 \end{bmatrix} \begin{Bmatrix} \delta \theta_1 \\ \delta \theta_2 \end{Bmatrix} = 0$$

#### For non trivial solution

$$\begin{vmatrix} k_2 & PL - k_2 \\ PL - k_1 - k_2 & k_2 \end{vmatrix} = 0$$

$$\Rightarrow P^2 L^2 - PL(k_1 + 2k_2) + k_1 k_2 = 0$$

$$\Rightarrow PL = \frac{k_1 + 2k_2 \pm \sqrt{k_1^2 + 4k_2^2}}{2}$$

*Eigenvalue equation* 

**Eigenvalues** 

We consider the case where  $k_1 = k_2 = k$ 

$$P_{cr} = \left(3 \pm \sqrt{5}\right) \frac{k}{2L} = \frac{2.618k}{L}, \frac{0.382k}{L}$$
 Eigenvalues

What this means is that when initial load is less than 0.382 k/L, a small disturbance will not disturb the equilibrium till the load becomes more than 0.382 k/L. When initial load is between 0.382 k/L and 2.618 k/L a small disturbance will not disturb the equilibrium till the load becomes more than 2.618 k/L. For practical purposes, only the lowest limit is important.

$$P_{cr} = 0.382 \frac{k}{L} \qquad \begin{array}{c} Eigenvalue \\ first \ mode \end{array}$$

$$k\delta\theta_1 + (PL - k)\delta\theta_2 \Rightarrow \begin{cases} \delta\theta_1 \\ \delta\theta_2 \end{cases} = \begin{cases} 1 - \frac{PL}{k} \\ 1 \end{cases} \delta\theta_2 = \begin{cases} 0.618 \\ 1 \end{cases} \delta\theta_2$$

$$P_{cr} = 2.618 \frac{k}{L} \qquad \begin{array}{c} Eigenvalue \\ second \ mode \end{array}$$

$$k\delta\theta_1 + (PL - k)\delta\theta_2 \Rightarrow \begin{cases} \delta\theta_1 \\ \delta\theta_2 \end{cases} = \begin{cases} 1 - \frac{PL}{k} \\ \delta\theta_2 \end{cases} \delta\theta_2 = \begin{cases} -1.618 \\ 1 \end{cases} \delta\theta_2$$

Substituting the critical values gives two eigenvectors, whose component ratios are the ratios of the deflections at those critical loads.