

# PHONONIC BRAGG REFLECTORS FOR THERMAL ISOLATION OF SEMICONDUCTOR QUBITS

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## ABSTRACT

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In recent years, the experimental capability in realizing and exercising full control over multiple coupled quantum dots in semiconductor heterostructures has advanced to the point where tuning the dots to the few-electron regime required for quantum computation applications will become impractical to perform manually, establishing the demand for automated procedures in this area. Moreover, charge stability diagrams, which represent a crucial tool in the tuning of quantum dot systems, become increasingly complex to understand and analyze due to the growing number of parameters influencing their structure. The goal of this thesis is to take a first step towards a fully computer-automated implementation of the tuning process by focusing on the coarse tuning of the system to the zero-electron regime.

I simulate charge diagrams of multiple coupled quantum dots in a linear array using a classical capacitive model, reproducing the honeycomb structure of double quantum dot charge diagrams. In order to automatically recognize the zero-electron regime in double-dot charge diagrams by identifying the charge transitions, I introduce a line detection algorithm using the Radon transform of the charge diagram able to detect lines even in noisy images by exploiting the fact that, in a suitable charge diagram detail, several transitions of the same kind and hence the same slope will show. The strong noise performance of the algorithm promises fast scan times during the tuning process. Finally, I present and test an algorithm that attempts to automatically tune a simulated double-dot system to the zero-electron regime using the line detection algorithm developed.

This is still  
the wrong  
abstract



## CONTENTS

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1	INTRODUCTION	3
1.1	Superlattice Structures for Thermal Isolation	3
1.2	Outline	5
2	THEORY	7
2.1	Elastic Properties of Materials	7
2.1.1	Stress Tensor	8
2.1.2	Strain Tensor	8
2.1.3	Stress-Strain Relations	9
2.2	Elastic Waves	10
2.2.1	Solution for Isotropic Materials	10
2.2.2	Elastic Wave Scattering at an Interface	12
2.2.3	Scattering at a System of Layers	17
2.2.4	Properties of Distributed Bragg Reflectors	21
3	IMPLEMENTATION	23
3.1	Code Structure and Functionality	23
3.1.1	Algorithms and their Scope of Application	24
3.1.2	Plots and Scripts for Evaluation	27
4	SIMULATIONS	29
5	CONCLUSION	31
	Bibliography	33
A	APPENDIX	35
A.1	Concrete Form of the Condition Matrices	35
A.2	Detailed Plots Picturing Scopes of Numerical Instabilities	35

## TODO LIST

---

<input type="checkbox"/>	This is still the wrong abstract . . . . .	iii
<input type="checkbox"/>	normalisation factor for angle integrations? . . . . .	4
<input type="checkbox"/>	Introduction to notation . . . . .	4
<input type="checkbox"/>	justification for modelling with elastic waves . . . . .	7
<input type="checkbox"/>	check if description still applies, citation? . . . . .	7
<input type="checkbox"/>	picture of stress definition? . . . . .	8
<input type="checkbox"/>	check definitions . . . . .	8
<input checked="" type="checkbox"/>	why is rotation irrelevant? . . . . .	8
<input checked="" type="checkbox"/>	what is the usual convention? . . . . .	9
<input type="checkbox"/>	mention of einstein convention? . . . . .	9
<input checked="" type="checkbox"/>	decomposition: e.g $a_L = \underline{u}(\underline{x}, t) \cdot \hat{k} * e^{-ikx+i\omega t}$ ? . . . . .	11
<input type="checkbox"/>	sketch . . . . .	11
<input checked="" type="checkbox"/>	introduced in 2.2.1 . . . . .	12
<input type="checkbox"/>	consistent mode description . . . . .	12
<input checked="" type="checkbox"/>	correct explanation? . . . . .	17
<input type="checkbox"/>	mention, that propagation of angles is needed first . . . . .	19
<input checked="" type="checkbox"/>	clear? . . . . .	19
<input type="checkbox"/>	Formeln in Theorieteil? . . . . .	23
<input type="checkbox"/>	reference to material parameters . . . . .	23
<input checked="" type="checkbox"/>	more specific explanation? . . . . .	25
<input type="checkbox"/>	mention $\Delta P$ . . . . .	27
<input type="checkbox"/>	Angular Plot reference . . . . .	27
<input type="checkbox"/>	overview plot reference . . . . .	27
<input checked="" type="checkbox"/>	Example Workflow still advantageous? . . . . .	28



## INTRODUCTION

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In recent years quantum computing technology has been of rising interest to scientists and the public. The early steps of exploiting the nature of quantum states by fabricating and controlling qubits [refs] are overcome and quantum supremacy, the supremacy of quantum computers over classical computers, is already achieved [ref]. With scalable concepts for many different implementations [refs] it is only a matter of time until quantum computers can be used commercially.

However, there are still a some hurdles to overcome before current setups are at this level. One of them is to reduce the decoherence time. It is crucial for the execution of large operations or algorithms and strongly depends on external conditions like for example, the influence of thermal radiation caused by photons or phonons[refs]. In solid state qubits, heat transfer by phonons is mostly enabled by electronics connected to the chip. This could be solved by a intermediate medium with very low thermal conductivity shielding the chip from thermal phonons.

### 1.1 SUPERLATTICE STRUCTURES FOR THERMAL ISOLATION

One possible approach to construct such a medium is by developing a superlattice structure that uses the effect of coherent phonon heat conduction. This concept exploits the properties of coherent phonons as coherent elastic waves and has proven to be effective for control of heat conduction at low temperatures [LuckyanovaGarg]. The underlying theory of this concept focuses on a special case of the macroscopic theory of heat conduction, where heat flow is explained as diffusive scattering of phonons between regions of different temperatures. However, the presented concept relies on interference effects of scattering at the constructed interfaces, so that such structures are significantly smaller than the phonon mean free path.

Further, such superlattices are known from optics as Distributed Bragg reflectors and usually consist of layer structures with alternating refraction indices. These are known to have so called spectral band gaps which are spectral regions where the wave is fully reflected. In optics this structure is usually used as highly efficient reflector for light from a defined angle.



The aim of this work will be to investigate practicable reflector configurations by developing a reusable simulation of elastic wave propagation through arbitrary multilayer structures. With this tool some configurations with different materials will be considered so that only an acceptable heat flux can pass the reflector. Based on the cooling power of current kryostats it is estimated, that the transmitted heat flux needs to be in the order of 10 mW.

In contrast to the optical case, the complete spectrum of thermal radiation for all incident angles  $\theta$  and polarisation modes  $i$  is relevant for the total expected heat flow through the reflector

$$J_z = \sum_{i=\{L,TV,TH\}} \int_0^{\omega_c} \int_{\theta,\phi} \mathcal{T}_i(\omega,\theta,\phi) g_i(\omega) n_B(\omega,T) \hbar\omega c_i \cos\theta d\omega d\theta d\phi \quad (1.1)$$

normalisation factor for angle integrations?

which will be justified later. The integrant consists of the transmittivity function  $\mathcal{T}_i(\omega,\theta,\phi)$  which is defined as proportion of transmitted intensity and can be interpreted here as transmission probability density.  $g_i(\omega)n_B(\omega,T)\hbar\omega$  is the spectral energy density per unit volume of the phononic thermal radiation with Bose distribution  $n_B$  and density of states  $g_i$ . Factoring this with the phonon velocity in z-direction  $c_i \cos\theta$  yields the spectral radiance of the uppermost medium which is the emitted power per frequency and spatial angle in z-direction. The exact form of  $g_i(\omega)$  and  $\omega_c$  is given by the Debye model which can be applied here.

As qubits are cooled to temperatures of less than 30mK, the temperature of the uppermost medium is also at low Kelvin temperatures. This justifies using the Debye model for the description of phonons which assumes linear dispersion  $\omega(k) = c_i|k|$ . For that reason, the calculation needs to be cut off at the Debye frequency  $\omega_D = (6\pi^2 n c_i^3)^{1/3}$  where  $n$  is the atomic density. In the following work the upper temperature is assumed to be  $T = 1.8\text{K}$  which is a temperature that can be achieved comfortably with current methods. With atomic densities of order  $10^{28} \frac{1}{\text{m}^3}$  and sound velocities of order  $10^3 \frac{\text{m}}{\text{s}}$ , the Debye frequency can approximated to be in the order of  $10^{13} \text{s}^{-1}$ . The spectral radiance for these frequencies at  $T = 1.8 \text{ K}$  is less than a thousandth of its maximum value, so that the present calculations will be performed with a lower, not material dependent cutoff of  $\omega_c = 3 \cdot 10^{12} \frac{1}{\text{s}}$ . In this frequency range the resulting wavelengths which are given by  $\lambda = \frac{2\pi c}{\omega}$ , are in the order of few nanometers, so that lattice properties of the materials can be neglected. Also, elastic anisotropies that may occur in crystal structures are simplified to an isotropic material with averaged properties.

#### Introduction to notation

- notation of vectors and tensors

## 1.2 OUTLINE



## THEORY

In order to understand the complexity of the problem and the utilised solving method, it is reasonable to revisit some fundamental concepts in the following chapter. At the same time, a consistent notation is introduced. At first, the tensor formalism for elastic properties of rigid bodies is introduced. After that, elastic waves are derived and analysed further. This discussion will be mostly based on [1], if not otherwise stated.

justification  
for modelling  
with elastic  
waves

It should be noted in advance, that vectors are denoted by a single underline and tensors or tensor fields of higher order by a number of underlines corresponding to their order. Also, Einstein summation convention should be applied if no explicit summation is given in the same formula.

check if de-  
scription still  
applies, cita-  
tion?

## 2.1 ELASTIC PROPERTIES OF MATERIALS

In the following, the formalism for bulk elastic properties of materials is introduced. As only small deformations are relevant for the considerations in this thesis, they are assumed to be in the regime of linear elastic behaviour, thus neglecting plastic or nonlinear behaviour.

In one dimension linear elastic behaviour can be described simply by Hooke's law, which states that a small displacement  $\Delta l$  caused by a force  $F$  is proportional to the force. Considering an object of length  $l$  and cross-sectional area  $A$ , **stress** and **strain** can be defined as

$$\text{Stress} \quad \sigma := \frac{F}{A} \quad \text{Strain} \quad \epsilon := \frac{\Delta l}{l} \quad (2.1)$$

Hooke's law can now be reformulated as

$$\sigma = C \epsilon \quad (2.2)$$

with **Young's Modulus C** as proportional constant.

### 2.1.1 Stress Tensor

For objects of finite size, stress can be defined locally on infinitesimal, cubic volume elements. These volume elements get deformed by forces that are applied to the object. If we consider a general force  $\Delta \underline{F}$  acting on a surface element  $\Delta A$  we can always divide this force into a normal component  $\Delta \underline{F}_n$  and two mutually perpendicular tangential components  $\Delta \underline{F}_{t1}$  and  $\Delta \underline{F}_{t2}$ . This implies the general definition of the stress tensor as

$$\sigma_{ij} = \frac{\text{force in direction } i}{\text{surface with normal in direction } j} \quad (2.3)$$

picture of stress definition?

where indices  $i$  and  $j$  denote one of the spatial directions  $x, y$  or  $z$ . If we assume the volume element to be in equilibrium, it follows that the normal forces on opposite sites and tangential forces on neighbouring sides equal each other. The latter implies that  $\sigma_{ij} = \sigma_{ji}$ . This leaves 6 independent components, the three normal stresses  $\sigma_{ii}$  and the three shear stresses  $\sigma_{ij}$ .

### 2.1.2 Strain Tensor

check definitions

Deformation of a three-dimensional object can be described using the displacement field  $\underline{u}(\underline{r}, t) := \underline{r}' - \underline{r}$ , which defines a displacement vector for each point  $\underline{r}$  in space in comparison to the deformed position  $\underline{r}'$ . Local stress only relates to change in displacement relative to neighbouring positions which allows to consider the Taylor expansion in first order

$$\underline{u}(\underline{r} + \Delta \underline{r}, t) = \underline{u}(\underline{r}, t) + \underline{\nabla} \underline{u}(\underline{r}, t) \Delta \underline{r} \quad (2.4)$$

with the Jacobian matrix  $(\underline{\nabla} \underline{u})_{ij} = \frac{\partial u_i}{\partial r_j}$ . This in turn can be decomposed into a symmetric part and an antisymmetric part. The symmetric part is defined as strain tensor:

$$\epsilon_{ij} = \frac{1}{2} \left( \frac{\partial u_i}{\partial r_j} + \frac{\partial u_j}{\partial r_i} \right) \quad (2.5)$$

why is rotation irrelevant?

It is a dimensionless measure for local deformation in contrast to the antisymmetric part, which represents local rotation.

$$xx \rightarrow 1 \quad yy \rightarrow 2 \quad zz \rightarrow 3 \quad yz = zy \rightarrow 4 \quad xz = zx \rightarrow 5 \quad xy = yx \rightarrow 6$$

Table 2.1: Voigt notation

### 2.1.3 Stress-Strain Relations

Having introduced the stress and the strain tensors, equation 2.2 can be generalised to three dimensions, thereby taking anisotropies of the material into account:

$$\sigma_{ij} = C_{ijkl} \epsilon_{kl} \quad \text{or} \quad \underline{\underline{\sigma}} = \underline{\underline{C}} \cdot \underline{\underline{\epsilon}} \quad (2.6)$$

This defines the **Elasticity tensor** as order 4 tensor with unit force / area. In general this tensor contains 81 components. However, both the stress

and the strain tensor are symmetric, so that  $C_{ijkl} = C_{jikl} = C_{ijlk}$ . This reduces the number of independent components to 36 and makes it possible to describe the Elasticity tensor as a  $6 \times 6$  matrix. For this representation, the so called Voigt notation maps pairs of coefficients of elasticity, strain and stress tensor to a single index as in table 2.1 This simplifies the notation of elasticity significantly, but it should be treated carefully. Tensors written in Voigt notation do not transform like vectors in each index.

One can also prove, that  $C_{ijkl} = C_{klij}$  considering elastic energy (see section 4.3.1 of [1]) which leads to 21 independent components. Further simplifications derive from crystal symmetry or limitations in anisotropy. In the following treatment, the material is assumed to be isotropic, which reduces the number of independent components to 2. These are typically introduced as Lamé constants  $\lambda$  and  $\mu$  defined by the relation

$$\sigma_{ij} = \lambda \delta_{ij} \epsilon_{kk} + 2\mu \epsilon_{ij} \quad (2.7)$$

which derives from equation 2.6 by regarding isotropy of the material [2]. From this equation the elasticity tensor can be expressed by a simplified  $6 \times 6$  matrix as in:

$$\underline{\underline{\sigma}} = \underline{\underline{C}} \cdot \underline{\underline{\epsilon}} = \begin{pmatrix} \lambda + 2\mu & \lambda & \lambda & 0 & 0 & 0 \\ \lambda & \lambda + 2\mu & \lambda & 0 & 0 & 0 \\ \lambda & \lambda & \lambda + 2\mu & 0 & 0 & 0 \\ 0 & 0 & 0 & 2\mu & 0 & 0 \\ 0 & 0 & 0 & 0 & 2\mu & 0 \\ 0 & 0 & 0 & 0 & 0 & 2\mu \end{pmatrix} \cdot \begin{pmatrix} \epsilon_1 \\ \epsilon_2 \\ \epsilon_3 \\ \epsilon_4 \\ \epsilon_5 \\ \epsilon_6 \end{pmatrix} \quad (2.8)$$

what is the usual convention?

mention of einstein convention?

## 2.2 ELASTIC WAVES

This formalism can now be used to introduce elastic waves. Their existence and behaviour depends on the equations of motion of the regarded medium. To obtain these, we consider at first a small volume  $\Delta V = \Delta x \Delta y \Delta z$  that is subject to a stress  $\sigma_{xx}(x)$  on the one side and to  $\sigma_{xx}(x + \Delta x)$  on the other side. The resulting net force becomes thus

$$\Delta F_x = [\sigma_{xx}(x + \Delta x) - \sigma_{xx}(x)] \Delta z \Delta y = \frac{\partial \sigma_{xx}}{\partial x} \Delta x \Delta y \Delta z \quad (2.9)$$

by approximating  $\sigma_{xx}(x + \Delta x)$  in first order and setting the frame of reference to the center of mass of the volume. The force leads to a displacement  $u_x$  of the volume in  $x$  direction and equals the product of the mass  $\rho \Delta x \Delta y \Delta z$  and acceleration in  $x$  direction  $\frac{\partial^2 u_x}{\partial t^2}$ . This yields the one dimensional partial differential equation

$$\rho \frac{\partial^2 u_x}{\partial t^2} = \frac{\partial \sigma_{xx}}{\partial x} \quad (2.10)$$

Assuming an isotropic material and using relation 2.6 we get

$$\rho \frac{\partial^2 u_x}{\partial t^2} = C_{11} \frac{\partial \epsilon_{xx}}{\partial x} = C_{11} \frac{\partial^2 u_x}{\partial x^2} \quad (2.11)$$

In an anisotropic medium the other stress components need to be taken into account leading to the general wave equation

$$\rho \frac{\partial^2 u_i}{\partial t^2} = \frac{\partial \sigma_{ij}}{\partial x_j} = C_{ijkl} \frac{\partial^2 u_l}{\partial x_j \partial x_k} \quad (2.12)$$

### 2.2.1 Solution for Isotropic Materials

In the following analysis, only isotropic media will be considered. In this case the isotropic stress-strain relation, equation 2.8, can be inserted in the wave equation which yields

$$\rho \frac{\partial^2 \underline{u}}{\partial t^2} = (\lambda + \mu) \underline{\nabla}(\underline{\nabla} \cdot \underline{u}) + \mu \underline{\nabla}^2 \underline{u} \quad (2.13)$$

By using the relation  $\underline{\nabla}^2 \underline{u} = \underline{\nabla}(\underline{\nabla} \cdot \underline{u}) - \underline{\nabla} \times (\underline{\nabla} \times \underline{u})$  for the vector Laplace operator one can express it as

$$\rho \frac{\partial^2 \underline{u}}{\partial t^2} = (\lambda + 2\mu) \underline{\nabla}(\underline{\nabla} \cdot \underline{u}) - \mu \underline{\nabla} \times (\underline{\nabla} \times \underline{u}) \quad (2.14)$$

If we now consider the Helmholtz decomposition of the displacement fields

$$\underline{u} = \underline{\nabla}\Phi + \underline{\nabla} \times \underline{\Psi} \quad (2.15)$$

with elastic potentials  $\Phi$  and  $\Psi$  and insert it into 2.14, it decouples to the two equations (see [3])

$$\frac{\partial^2 \Phi}{\partial t^2} = c_L^2 \underline{\nabla}^2 \Phi \quad (2.16)$$

$$\frac{\partial^2 \underline{\Psi}}{\partial t^2} = c_T^2 \underline{\nabla}^2 \underline{\Psi} \quad (2.17)$$

$$\text{with } c_L = \left( \frac{\lambda + 2\mu}{\rho} \right)^{1/2} \quad \text{and} \quad c_T = \left( \frac{\mu}{\rho} \right)^{1/2} \quad (2.18)$$

Those equations represent four wave equations for the elastic potentials with phase velocities  $c_L$  and  $c_T$ .

In analogy to electromagnetic waves we can now make the harmonic plane wave ansatz  $\Phi(\underline{x}, t) = u_L e^{i(\underline{k}_L \underline{x} - \omega t)}$  and  $\underline{\Psi}(\underline{x}, t) = \underline{u}_T e^{i(\underline{k}_T \underline{x} - \omega t)}$  with arbitrary amplitudes  $u_L \in \mathbb{C}$  and  $\underline{u}_T \in \mathbb{C}^3$ . Here, the vector  $\underline{k} = \frac{2\pi}{\lambda}$  denotes the wave vector defined by the wavelength  $\lambda$  and  $\omega$  is the circular frequency defined by the oscillation frequency  $f$  with  $\omega = 2\pi f$ . The differentiation between  $\underline{k}_L$  and  $\underline{k}_T$  is necessary because of the different phase velocities for the elastic potentials resulting in different wave numbers according to  $c = \frac{\omega}{|\underline{k}|}$ .

Substituting the wave ansätze into equation 2.15, we get

$$\underline{u} = u_L \underline{k} e^{i(\underline{k}_L \underline{x} - \omega t)} + \underline{k} \times \underline{u}_T e^{i(\underline{k}_T \underline{x} - \omega t)} \quad (2.19)$$

Here we can see now, that the elastic potential  $\Phi$  is responsible for waves with longitudinal polarisation and  $\underline{\Psi}$  for waves with transversal propagation. Furthermore we can identify an orthonormal polarisation basis depending on the propagation direction  $\hat{\underline{k}} = \frac{\underline{k}}{|\underline{k}|}$  so that  $\underline{u}$  decomposes to

$$\underline{u} = a_L \underline{p}_L e^{i(\underline{k}_L \underline{x} - \omega t)} + (a_{TH} \underline{p}_{TH} + a_{TV} \underline{p}_{TV}) e^{i(\underline{k}_T \underline{x} - \omega t)} \quad (2.20)$$

In general, the polarisation vector for longitudinal polarisation  $\underline{p}_L$  is fixed to be  $\hat{\underline{k}}$ . If scattering at an interface is considered, a plane of incidence can be defined that is spanned by  $\hat{\underline{k}}$  and the normal vector of the interface. The transversal polarisation vectors are then defined as transversal horizontal polarisation  $\underline{p}_{TH}$  pointing out of the plane of incidence and transversal vertical polarisation  $\underline{p}_{TV}$  lying in plane of incidence orthonormal to  $\hat{\underline{k}}$ , here now defined as  $\underline{p}_{TV} = \underline{p}_L \times \underline{p}_{TH}$ .

decomposition:  
e.g  
 $a_L = \underline{u}(\underline{x}, t) \cdot \hat{\underline{k}} * e^{-i\hat{\underline{k}}\underline{x} + i\omega t}$   
?

sketch



To sum up, those polarisation vectors define three polarisation modes L, TH and TV denoting longitudinal, transversal horizontal and transversal vertical polarisation which can be superimposed arbitrarily. The coefficients  $a_i$  may be complex to express an additional phase between the components. However, it should be noted, that this is merely convenient for calculation and that the physical wave behaves like the real part of the shown equations.

### 2.2.2 Elastic Wave Scattering at an Interface

#### Boundary Conditions

On the way to describe wave propagation through complex layer structures it is useful to consider a single interface first. In the upper half space  $S_1$  and lower half space  $S_2$  we assume homogeneous materials with different elastic constants and thus different sound velocities. At the interface, which is chosen to be the xy-plane the elastic properties are discontinuous. However, we can assume that those layers connected strongly so that the displacement field and normal stresses are continuous at the boundary [4, pp. 182, 185]. This leads to the boundary conditions:

$$\underline{u}^{(1)}(\underline{r}, t)|_{\underline{r} \in \partial S_1} = \underline{u}^{(2)}(\underline{r}, t)|_{\underline{r} \in \partial S_2} \quad (2.21)$$

$$\sigma_{13}^{(1)}(\underline{r}, t)|_{\underline{r} \in \partial S_1} = \sigma_{13}^{(2)}(\underline{r}, t)|_{\underline{r} \in \partial S_2} \quad (2.22)$$

$$\sigma_{23}^{(1)}(\underline{r}, t)|_{\underline{r} \in \partial S_1} = \sigma_{23}^{(2)}(\underline{r}, t)|_{\underline{r} \in \partial S_2} \quad (2.23)$$

$$\sigma_{33}^{(1)}(\underline{r}, t)|_{\underline{r} \in \partial S_1} = \sigma_{33}^{(2)}(\underline{r}, t)|_{\underline{r} \in \partial S_2} \quad (2.24)$$

In these six equations displacement and stress from upper and lower medium are differentiated by the given superscript index.

#### Law of Refraction

If we now consider a plane wave that is incident on that interface, we can define a plane of incidence as in section 2.2.1 and rotate the coordinate system so that it equals the yz-plane which simplifies the polarisation basis. This leaves the physical problem only invariant, if we assume an isotropic material as stated before. In that setting, the wave vector  $\underline{k}_i$  for a particular mode  $i \in \{L, TV, TH\}$  can be parametrised by the angle of incidence  $\theta$  as

$$\underline{k}_i = \frac{\omega}{c_i} (0, \sin(\theta_i), -\cos(\theta_i))^T \quad (2.25)$$

introduced in  
2.2.1

consistent  
mode descrip-  
tion

The polarisation basis can then be defined as

$$\underline{p}_L = \hat{k} = (0, \sin(\theta_i), -\cos(\theta_i))^T \quad (2.26)$$

$$\underline{p}_{TH} = (1, 0, 0)^T \quad (2.27)$$

$$\underline{p}_{TV} = \underline{p}_L \times \underline{p}_{TH} = (0, -\cos(\theta), -\sin(\theta))^T \quad (2.28)$$

It is now possible to extract a law of refraction from the boundary conditions in equation 2.21 (see [4, 168ff]). A simple approach is to consider scattering of the transversal horizontal mode (TH) so that the displacement is only in x-direction. An incoming plane wave with wave vector  $\underline{k}_{in}$  is partially reflected to a wave along  $\underline{k}_r$  and partially transmitted through the interface to a wave with  $\underline{k}_t$ . Also, different frequencies  $\omega_i$  are assumed for the scattered waves so that the wave in the upper medium is

$$u_1^{(1)} = a_{in} e^{i(k_{in} \sin \theta_{in} y - \omega_{in} t)} + a_r e^{i(k_r (\sin \theta_r y - \omega_r t))} \quad (2.29)$$

and in the lower medium

$$u_1^{(2)} = a_t e^{i(k_t \sin \theta_t y - \omega_t t)} \quad (2.30)$$

Here it was used, that the interface is at  $z = 0$  which simplifies the following derivation. According to the boundary conditions, these displacements must equal each other at any time and for all points on the interface. This leads to the immediate conclusion, that

$$\omega = \omega_{in} = \omega_r = \omega_t \quad (2.31)$$

After removing the common factor  $e^{-i\omega t}$  from the equation and expressing the wave numbers by frequency and sound velocity as  $k = \frac{\omega}{c}$ , we get

$$a_{in} e^{i \frac{\omega}{c_{T,1}} \sin \theta_{in} y} + a_r e^{i \frac{\omega}{c_{T,1}} \sin \theta_r y} = a_t e^{i \frac{\omega}{c_{T,2}} \sin \theta_t y} \quad (2.32)$$

This however can be only fulfilled for all  $y \in \mathbb{R}$  if the relations

$$a_{in} + a_r = a_t \quad (2.33)$$

$$\frac{\sin \theta_{in}}{c_{T,1}} = \frac{\sin \theta_r}{c_{T,1}} = \frac{\sin \theta_t}{c_{T,2}} \quad (2.34)$$

are met. Equation 2.34 can be also interpreted as the equality of the wave vector component parallel to the interface  $k_y = \sin \theta_i \frac{\omega}{c_i}$ .

The same procedure can be applied to the y and z component of the displacement field, where transversal vertical (TV) and longitudinal (L) modes need to be considered [4, p. 185]. However, one significant difference is that even for a

single incident mode, both  $L$  and  $TV$  mode are possible after being reflected or transmitted. This will only add a term of the form  $a_i e^{i\frac{\omega}{c} \sin\theta_i y}$  to each side of equation 2.32, but will result in analogous relations to 2.31 and 2.34, namely the invariance of frequency  $\omega$  and  $k_2$  of each mode under scattering at the interface.

To conclude, the angle  $\theta_{out}$  of any outgoing polarisation mode can be determined from the incoming wave's angle  $\theta_{in}$  and the sound velocities for incoming and outgoing wave  $c_{in}, c_{out}$  by

$$\frac{\sin\theta_{in}}{c_{in}} = \frac{\sin\theta_{out}}{c_{out}} \quad (2.35)$$

This can be transferred to the optical law of snellius by introducing an effective refraction index  $n_i = \frac{1}{c_i}$ . Consequently, TH-polarised waves scatter like electromagnetic waves and for example the reflected angle is always the incident angle. The main difference to optics becomes clear for L and TV polarised waves. Scattering within the same polarisation mode is the same as for the TH mode, but now those modes can scatter into each other. Equation 2.35 still applies, but energy is also emitted to additional modes as shown in figure 2.1.

It is also possible to get the analytical solutions for refraction of a single mode at an interface by evaluating the remaining boundary conditions and the results from continuity of displacement. The results are usually expressed by defining a reflection coefficient  $r_i$  and a transmission coefficient  $t_i$  for participating mode  $i \in \{L, TH, TV\}$  as

$$r_i := \frac{a_{r,i}}{a_{in,i}} \quad (2.36)$$

$$t_i := \frac{a_{t,i}}{a_{in,i}} \quad (2.37)$$

which usually have a functional dependency on the incident angle. With introducing the impedance  $Z(\theta)$  of mode  $i$  of a medium as

$$Z_i(\theta) = \frac{\rho c_i}{\cos\theta} \quad (2.38)$$

it is possible to obtain the simple relations

$$r_{TH} = \frac{Z_{in,TH} - Z_{out,TH}}{Z_{in,TH} + Z_{out,TH}} \quad (2.39)$$

$$t_{TH} = \frac{2Z_{in,TH}}{Z_{in,TH} + Z_{out,TH}} \quad (2.40)$$

describing the refraction of  $TH$  polarisation from analogous calculations with electromagnetic waves [5, pp. 46, 14]. Analytic solutions for the remaining modes are much more complex and can be looked up in [6, p. 83].

### Total Internal Reflection

Total internal reflection is known as the phenomenon when a wave is totally reflected at a boundary for incident angles greater than a critical angle  $\theta_{tot}$ . If  $c_1$  and  $c_2$  are the phase velocities first and second medium, the outgoing angle

$$\theta_2 = \arcsin\left(\frac{c_1}{c_2} \sin(\theta_1)\right) \quad c_2 < c_1 \quad (2.41)$$

has no real solution for  $\theta_1 > \theta_{tot} = \arcsin \frac{c_2}{c_1}$ . There is however a complex solution and this resulting complex angle can be interpreted in the given wave formalism [5, p. 5]. This interpretation also allows to extend the concept to two arbitrary polarisation modes that the wave can scatter inbetween, regardless of propagation direction.

Considering the case  $\theta_1 > \theta_{tot}$ ,  $\theta_2$  can be expressed as  $\theta_2 = \frac{\pi}{2} + i\alpha$ ,  $\alpha \in \mathbb{R}$ . Inserting this into the general wave solution equation 2.20, we need to calculate the wave vector  $\underline{k}$  as in equation 2.25. This results in

$$k_y = k \sin \theta_2 = k \cosh \alpha \quad k_z = k \cos \theta = ik \sinh \alpha \quad (2.42)$$

This complex wave vector inserted into the factor  $e^{ik_z z}$  results in  $e^{-k \sinh \alpha z}$  which represents an evanescent wave. This type of wave is attenuated exponentially so that the wave behaves as expected.

In case of frustrated total internal reflection this exponential behaviour becomes relevant, when another interface is close to the first reflecting interface. Then it is possible for the usually totally reflected mode to transmit intensity into a mode of the third medium.

### Conservation of Energy

Since acoustic waves transport energy, we can define an intensity for them and consider the conservation of energy at an interface. According to [4, p. 166] the transmitted time averaged power per unit area of a transversal elastic wave in a homogeneous medium is

$$I_L = \frac{1}{2}(\lambda + 2\mu) \frac{\omega^2}{c_L} |a_L|^2 \quad (2.43)$$

and similarly for a transversal elastic wave of a single polarisation mode

$$I_T = \frac{1}{2} \mu \frac{\omega^2}{c_T} |a_T|^2 \quad (2.44)$$

where  $a_T$  is either  $a_{TV}$  or  $a_{TH}$  depending on the regarded mode. This quantity is also known as time averaged intensity. By using the definition of  $c_L$  and  $c_T$  from equation 2.18 we get the more general relation

$$I_p = \frac{\rho\omega^2}{2} |a_p|^2 c_p \quad (2.45)$$

with  $p \in \{L, TH, TV\}$  denoting the polarisation mode. It should be noted that the referred unit area is perpendicular to the direction of propagation.

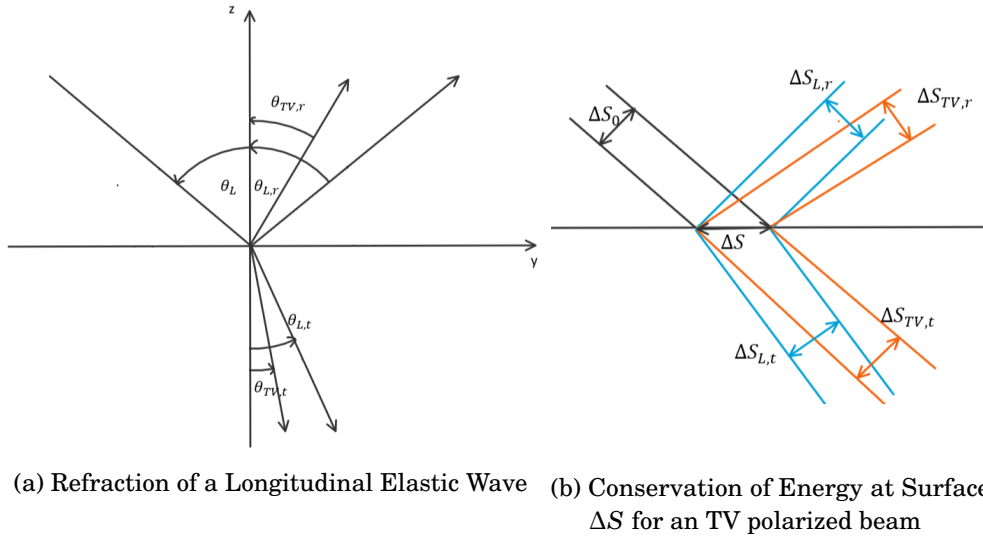


Figure 2.1

In the setting of an arbitrary polarised wave scattered at an interface, there are six possible outgoing modes as discussed previously. To get an energy relation between the intensities of each participating mode we consider the energy balance of a surface element  $\Delta S$  of the interface, on which an arbitrarily polarised beam with cross-sectional area  $\Delta S_0$  and intensity  $I_{in}$  is incident. The cross-sectional areas of the outgoing beams are named according to figure 2.1b which can be also transferred to the case of an incident TH polarised wave.

The power each beam transmits or emits is given by the product  $P_p = I_p \Delta S_p$ .  $\Delta S_p$  is related to  $\Delta S$  by  $\Delta S_p = \cos\theta_p \Delta S$ ,  $p$  denoting any of the depicted beams and  $\theta_p$  being the according angle.

In conclusion, the energy balance results in

$$I_{in} \cos\theta_{in} = \sum_{i=L,TH,TV} I_{i,r} \cos\theta_{i,r} + I_{i,t} \cos\theta_{i,t} \quad (2.46)$$

From that we can divide by  $I_{in} \cos \theta_{in}$  and define reflectivity  $\mathcal{R}$  and transmittivity  $\mathcal{T}$  of the interface as fraction of reflected and transmitted power by

$$\mathcal{R} := \frac{I_{L,r} \cos \theta_{L,r} + I_{TH,r} \cos \theta_{T,r} + I_{TV,r} \cos \theta_{T,r}}{I_{in} \cos \theta_{in}} \quad (2.47)$$

$$\mathcal{T} := \frac{I_{L,t} \cos \theta_{L,t} + I_{TH,t} \cos \theta_{T,t} + I_{TV,t} \cos \theta_{T,t}}{I_{in} \cos \theta_{in}} \quad (2.48)$$

so that  $\mathcal{R} + \mathcal{T} = 1$  is fulfilled.

This can be easily extended to multi layer structures replacing the single interface by the multi layer system so that the transmitted intensities are the intensities that propagated through the whole structure.

The relation  $\mathcal{T} + \mathcal{R} = 1$  can then be used to verify the obtained results for any layer structure.

### 2.2.3 Scattering at a System of Layers

The considerations of one interface can now be extended to a system of  $N$  interfaces as shown in figure 2.2. Each layer gets an index  $n$  starting from the uppermost layer of index  $n = 0$ . The first interface from the top is still located at  $z_1 = 0$  and has index  $n = 1$ . The subsequent interfaces follow below at depths  $z_n$ . Their depth depends on the intermediate layer thicknesses  $d_n = z_n - z_{n+1}$  with layer index  $n$ . Similar as before, we assume now the displacement field in each layer  $n$  as

$$\underline{u}^{(n)}(\underline{x}, t) = \sum_{i \in \{L, TH, TV\}} t_{n,i} \underline{p}_{n,i}^t e^{i(\underline{k}_{n,i}^t \underline{x} - \omega t)} + r_{n,i} \underline{p}_{n,i}^r e^{i(\underline{k}_{n,i}^r \underline{x} - \omega t)} \quad (2.49)$$

This formula is a superposition of a down going wave with quantities denoted by  $t$  and an upgoing wave caused by reflections with quantities denoted by  $r$ . The upgoing  $\underline{k}^r$  is flipped in  $z$ -direction compared to  $\underline{k}^t$ . Further, the complex amplitudes of down and upgoing modes are directly denoted by  $t_{n,i}$  and  $r_{n,i}$ . As the angle  $\theta_{n,i}$  between  $\underline{k}$  and  $z$ -axis is changes by scattering, also  $\underline{k}_{n,i}$  and polarisation vector  $\underline{p}_{n,i}$  is different for each material.

To evaluate the transmission through the stack of layers, it is the task is now to calculate the coefficients  $t_{N,i}$  and  $r_{1,i}$  given coefficients  $t_{1,i}$  and with the assumption  $r_{N,i} = 0$  for all  $i \in \{L, TH, TV\}$ . This assumption can be made because we only consider the incoming power on the chip. Thermal radiation from the chip itself is significantly lower We further choose exactly one  $t_{1,i} = 1$  to mark the incident mode. This simplifies solving the problem and lets the coefficients  $t_{N,i}$  and  $r_{1,i}$  become the transmission and reflection coefficients of the layer system.

correct explanation?

For this, all  $6N$  boundary conditions need to be solved. These are constructed by inserting the general displacement field of each participating layer as in equation 2.49 in the boundary conditions for a single interface (equation 2.21). As a result each displacement field of an enclosed layer is evaluated twice, once at the upper boundary and once at the lower boundary. This results in a phase shift of  $e^{-ik_z d_n}$  between upper and lower displacement.

As many systems of equations this problem can be formulated with matrices. In the following, two solution methods are introduced. In the later course, both methods will be compared to each other on the base of practical advantages and disadvantages.

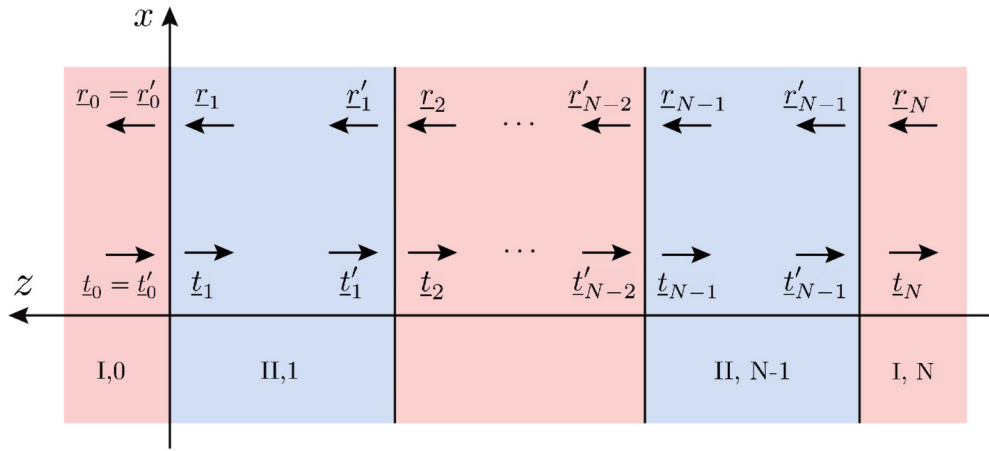


Figure 2.2: Geometry and amplitude nomenclature of a N layer strong bragg reflector

### Transfer Matrix Method

To introduce the matrix formalism of the Transfer Matrix Method (TMM) it is useful to focus on a single interface with index  $n$  between layers  $n-1$  and  $n$  located at depth  $z_n$ . We can define a generalised state vector  $\underline{v}_i$  in each layer  $i$  as [7]

$$\underline{v}_i = (u_1^{(i)}, u_2^{(i)}, u_3^{(i)}, \sigma_{13}^{(i)}, \sigma_{23}^{(i)}, \sigma_{33}^{(i)})^T \quad (2.50)$$

Each of the components depends on location  $\underline{x}$  and time  $t$  in the layer. We also introduce the superscript "+" for evaluating an arbitrary multidimensional function of space and layer index  $n$  at interface  $\mathcal{F}_n$  and superscript "-" for evaluating at  $\mathcal{F}_{n-1}$  so that the boundary conditions at the  $n$ -th interface  $\mathcal{F}_n$  are

$$\underline{v}_n^+ = \underline{v}_{n+1}^- \quad (2.51)$$

In addition to equation 2.49, the stress components in  $\underline{v}$  can be evaluated as well. From stress strain relation 2.6 we get for an isotropic material

$$C_{1313}^{(n)} \epsilon_{13}^{(n),+} = C_{1313}^{(n+1)} \epsilon_{13}^{(n+1),-} \quad (2.52)$$

$$C_{2323}^{(n)} \epsilon_{23}^{(n),+} = C_{2323}^{(n+1)} \epsilon_{23}^{(n+1),-} \quad (2.53)$$

$$\sum_{i=1}^3 C_{33ii}^{(n)} \epsilon_{ii}^{(n),+} = \sum_{i=1}^3 C_{33ii}^{(n+1)} \epsilon_{ii}^{(n+1),+} \quad (2.54)$$

Furthermore, the strain can be evaluated by inserting the general displacement field 2.49 into its definition  $\epsilon_{ij} = \frac{1}{2} \left( \frac{\partial u_i}{\partial r_j} + \frac{\partial u_j}{\partial r_i} \right)$ .

mention, that propagation of angles is needed first

At this point it is necessary to know all angles at which the different modes propagate through each layer in order to construct the equations as linear relations. These can be obtained by following the considerations of section 2.2.2. Because of our limitation to one incident mode on the top layer, also only one propagation angle per distinct phase velocity in the medium is possible. From the structure of  $\underline{u}^{(n)}$  it is possible to construct a  $6 \times 6$  matrix  $\underline{M}_n$  so that

$$\underline{v}_n(z) = \underline{\underline{M}}_n \underline{\underline{P}}_n(z - z_n) \underline{a}_n \quad (2.55)$$

. This step is considered in detail in the appendix. The coefficient vector  $\underline{a}_n$  of layer  $n$  is defined as

$$\underline{a}_n = (\underline{t}_n, \underline{r}_n)^T = (t_{n,L}, t_{n,TH}, t_{n,TV}, r_{n,L}, r_{n,TH}, r_{n,TV})^T \quad (2.56)$$

containing the wave amplitudes at the upper edge of each layer  $z = z_n$ . As an exception,  $r_{0,i}$  and  $t_{0,i}$  are defined directly at the first interface.

$\underline{\underline{P}}_n(z)$  represents the phase factors due to wave propagation as diagonal matrix

$$\underline{\underline{P}}_n(z) = \text{diag}\{e^{ik_{z,n}^{(L)}z}, e^{ik_{z,n}^{(TH)}z}, e^{ik_{z,n}^{(TV)}z}, e^{-ik_{z,n}^{(L)}z}, e^{-ik_{z,n}^{(TH)}z}, e^{-ik_{z,n}^{(TV)}z}\} \quad (2.57)$$

The factor  $e^{k_{y,n}^{(i)}y}$  was removed from the expression as it does not change at the boundary. This automatically sets the phase of the displacement wave to zero at the upper boundary of the layer. The coefficients at the lower boundary are in the following denoted by  $\underline{a}'_n = \underline{\underline{P}}_n(-d_n) \underline{a}_n$ .

clear?

As a result the boundary conditions at an individual interface between layers  $n - 1$  and  $n$  can be written as

$$\underline{\underline{M}}_{n-1} \underline{a}'_{n-1} = \underline{\underline{M}}_n \underline{a}_n \quad (2.58)$$



If we assume that  $\underline{\underline{M}}_n$  is invertible, we can define the transfer matrix  $S_n$  by

$$\underline{a}_n = \underline{\underline{M}}_n^{-1} \underline{\underline{M}}_{n-1} \underline{a}'_{n-1} = \underline{S}_n \underline{a}'_n \quad (2.59)$$

which enables us to obtain the wave coefficients after scattering by a simple matrix multiplication.

With this knowledge we can assemble a global transfer matrix for all interfaces by

$$\underline{\underline{S}} = \underline{\underline{S}}_N \cdot \underline{\underline{P}}_{\underline{\underline{N}}}(z_N - z_{N-1}) \cdot \underline{\underline{S}}_{N-1} \cdot \underline{\underline{P}}_{\underline{\underline{N-1}}}(z_{N-1} - z_{N-2}) \cdots \underline{\underline{S}}_2 \cdot \underline{\underline{P}}_{\underline{\underline{2}}}(z_2 - z_1) \cdot \underline{\underline{S}}_1 \quad (2.60)$$

so that the final wave coefficients  $\underline{a}_N$  below the lowest interface are

$$\begin{pmatrix} \underline{t}_n \\ \underline{r}_n \end{pmatrix} = \underline{S} \cdot \begin{pmatrix} \underline{t}_1 \\ \underline{r}_1 \end{pmatrix} = \begin{pmatrix} \underline{\underline{T}} & \underline{\underline{C}} \\ \underline{\underline{C}} & \underline{\underline{R}} \end{pmatrix} \cdot \begin{pmatrix} \underline{t}_1 \\ \underline{r}_1 \end{pmatrix} \quad (2.61)$$

Remembering that  $\underline{r}_n = 0$ , we can solve for  $\underline{r}_1$  and  $\underline{t}_n$

$$\underline{r}_1 = -\underline{\underline{R}}^{-1} \underline{\underline{C}} \underline{t}_1 \quad (2.62)$$

$$\underline{t}_n = \begin{pmatrix} \underline{\underline{T}} & \underline{\underline{C}} \\ -\underline{\underline{R}}^{-1} \underline{\underline{C}} & \underline{\underline{R}} \end{pmatrix} \cdot \begin{pmatrix} \underline{t}_1 \\ \underline{r}_1 \end{pmatrix} \quad (2.63)$$

### Linear System of Equations

As an alternative, a more direct solving approach is presented by solving the system of linear equations with the inversion of a single matrix. This method will be called from now on LSE-method (Linear System of Equations).

For this method the coefficients for upgoing waves are defined to have their phase origin at the lower interface instead of the upper interface as defined in 2.57, so that the P-matrix becomes

$$\underline{\underline{P}}_n(z) = \text{diag}\{e^{ik_{z,n}^{(L)}z}, e^{ik_{z,n}^{(TH)}z}, e^{ik_{z,n}^{(TV)}z}, e^{-ik_{z,n}^{(L)}(z+d_n)}, e^{-ik_{z,n}^{(TH)}(z+d_n)}, e^{-ik_{z,n}^{(TV)}(z+d_n)}\} \quad (2.64)$$

. The reason is that the diagonal entries evaluated at the layer boundaries  $z = 0$  and  $z = -d_n$  will now have always absolute values of 1 or lower. The latter is true for total internal reflection in contrary to  $\underline{\underline{P}}$  from TMM.

For better clarity in notation we can rewrite matrix  $\underline{\underline{M}}$  as composition of two  $6 \times 3$  matrices  $\underline{\underline{M}} = (\underline{\underline{T}}, \underline{\underline{R}})$  and  $\underline{\underline{P}}$  as block diagonal of two  $3 \times 3$  matrices  $\underline{\underline{P}} = \text{diag}\{\underline{\underline{D}}_{\underline{\underline{n}}}(z), \underline{\underline{D}}_{\underline{\underline{n}}}(-(z+d_n))\}$ . The diagonal matrix  $\underline{\underline{D}}_{\underline{\underline{n}}}$  is here defined as a  $3 \times 3$  diagonal matrix with  $\underline{\underline{D}}_{\underline{\underline{n}}}(z) = \text{diag}\{e^{ik_{z,n}^{(L)}z}, e^{ik_{z,n}^{(TH)}z}, e^{ik_{z,n}^{(TV)}z}\}$

The boundary conditions at interface  $n$ , below layer  $n$  are then

$$\left( \underline{T}_n, \underline{R}_n \cdot \underline{D}_n(-d_n) \right) \begin{pmatrix} \underline{t}_n \\ \underline{r}_n \end{pmatrix} = \left( \underline{T}_{n-1} \cdot \underline{D}_{n-1}(-d_{n-1}), \underline{R}_{n-1} \right) \begin{pmatrix} \underline{t}_{n-1} \\ \underline{r}_{n-1} \end{pmatrix} \quad (2.65)$$

With  $\underline{T}'_{n-1} = \underline{T}_{n-1} \cdot \underline{D}_{n-1}(-d_{n-1})$  and analogously for  $\underline{R}'_n$ , this is equivalent to

$$\left( \underline{T}'_{n-1}, \underline{R}_{n-1} - \underline{T}_n, -\underline{R}'_n \right) \begin{pmatrix} \underline{t}_{n-1} \\ \underline{r}_{n-1} \\ \underline{t}_n \\ \underline{r}_n \end{pmatrix} = 0 \quad (2.66)$$

This denotes now a  $6 \times 12$  matrix for a single interface. In general the total system is constructed as  $6N \times 6N$  matrix

$$\begin{pmatrix} \underline{R}_0 & -\underline{T}_1 & -\underline{R}'_1 & 0 & 0 & \cdots & 0 \\ 0 & \underline{T}'_1 & \underline{R}_1 & -\underline{T}_2 & -\underline{R}'_2 & \cdots & \vdots \\ \vdots & \ddots & \ddots & \ddots & \ddots & \ddots & \vdots \\ \vdots & \ddots & \ddots & \ddots & \ddots & \ddots & \vdots \\ 0 & \cdots & \cdots & 0 & \underline{T}'_{N-1} & \underline{R}_{N-1} & -\underline{T}_N \end{pmatrix} \begin{pmatrix} \underline{r}_0 \\ \underline{t}_1 \\ \underline{r}_1 \\ \vdots \\ \underline{t}_N \end{pmatrix} = \begin{pmatrix} -\underline{T}_0 \underline{t}_0 \\ 0 \\ \vdots \\ \vdots \\ \underline{R}_N \underline{r}_N \end{pmatrix} \quad (2.67)$$

The transfer of  $\underline{T}_0 \underline{t}_0$  and  $\underline{R}_N \underline{r}_N$  is necessary to ensure quadratic shape of the matrix which is necessary for inversion. However, this limits the number of interfaces to  $N > 1$ . By setting  $\underline{r}_N = 0$  as before we obtain  $\underline{r}_0$  and  $\underline{t}_N$  after inverting the  $6N \times 6N$  matrix.

#### 2.2.4 Properties of Distributed Bragg Reflectors

In case of normal incidence on the reflector, the modes L and TV do not mix and the relations of transmission and reflection become analogous to optical waves. This can be seen from 2.35 and the fact, that a displacement normal to the interface does not produce a parallel displacement and the other way round. Therefore, fundamental properties of Distributed Bragg Reflector for optical waves apply here as well.

For the following considerations, many optic based sources are reinterpreted as acoustic quantities. The already mentioned concept of impedance is also applicable to electromagnetic waves by defining it as the ratio of electric magnetic field amplitude in the medium [5]. This evaluates to

$$Z = \sqrt{\frac{\mu_0 \mu_r}{\epsilon_0 \epsilon_r}} = Z_0 \frac{\mu_r}{n} \quad (2.68)$$

with vacuum permeability  $\mu_0$  and vacuum permittivity  $\epsilon_0$  and the relative permeability and permittivity denoted by  $r$ .  $Z_0$  is the vacuum impedance. It is usual to assume  $\mu_r = 1$  as most regarded materials are non magnetic, so that  $Z = Z_0/n$  is valid. As a result, relations for optical Bragg reflectors at normal incidence can be easily converted to relations for acoustic waves under normal incidence.

The typically used DBR structure consists of a repeated unit cell of two layers with impedances  $Z_{1,i}$  and  $Z_{2,i}$  for different modes  $i$ . The thicknesses are often chosen to have a thickness of a quarter wavelength of a selected frequency mode to be reflected. With sufficient number of repetitions of this unit cell, spectral band gaps form, where waves with the selected frequency and a certain interval around it are reflected with high reflectivities. Moreover, the described setup within the unit cell proves to maximize the spectral gap width to mid gap frequency ratio  $\frac{\Delta f}{f_0}$  [8].

In a more general setup, where the single layers do not have exactly the thickness of a quarter wavelength of a selected frequency mode, there will be still a development of frequency stop bands. This is because of the periodicity of the unit cell. For electromagnetic waves the stop bands form around the bragg wavelength

$$m\lambda_B/2 = n_1d_1 + n_2d_2 \quad m = 1, 2, 3, \dots \quad (2.69)$$

with refraction indices  $n_i = \frac{c_0}{c_i}$ . To get an analogous relation for elastic waves, we can simply replace the vacuum speed of light  $c_0$  by an arbitrary sound velocity. If  $\lambda_B$  is then expressed as bragg frequency  $f_B = \frac{c_0}{\lambda_B}$  we get a similar result with known quantities as

$$2\frac{f_B}{m} = \left(\frac{d_1}{c_1} + \frac{d_2}{c_2}\right)^{-1} = \left(\frac{\rho_1d_1}{Z_1} + \frac{\rho_2d_2}{Z_2}\right)^{-1} \quad (2.70)$$

which is also valid for acoustic waves. In addition to the described position of the bandgaps, also their width can be derived. It appears, that the spectral gap width to bragg frequency ratio of the first bandgap can be expressed as

$$\frac{\Delta f}{f_B} = \frac{4}{\pi} \arcsin\left(\frac{|Z_1 - Z_2|}{Z_1 + Z_2}\right) \quad (2.71)$$

These relations can be used to verify

## IMPLEMENTATION

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On basis of the preceding theory a full simulation of acoustic wave propagation through multilayer structures was developed. This will be described in more detail in the following aiming to provide a short introduction into code structure and functionality. Arisen challenges and their solution will be presented afterwards.

### 3.1 CODE STRUCTURE AND FUNCTIONALITY

The main components of the project are represented by the three python modules `dbr.py`, `materials.py` and `plotfunctions.py` to organise written methods by purpose.

`materials.py` contains elastic parameters of materials that were considered for usage in a reflector structure. Each entry is a python function that returns density  $\rho$  and elastic constants  $\lambda$  and  $\mu$ . Most databases accessible through the internet contain different elastic constants like Young's Modulus or Poisson's ratio, so that those are converted with help of a formula table as in [2, p. 30]. The exact materials and their sources are listed in table

The file `dbr.py` contains the most important simulation tools. Those functions are structured in two classes, `Reflector` and `Layer` to reduce the number of parameters to be passed to the next method.

The main idea for the `Layer` class is to wrap the important material parameters  $\lambda$ ,  $\mu$  and  $\rho$  passed from a material parameter function and to convert them internally to sound velocities  $c_i$  and elastic tensor. In addition, each object stores its thickness and material name. The latter is useful for fast identification of the layer material and debugging outputs. Also, it contains several methods for calculation of Layer specific quantities. One of those quantities is `getConditionMatrix()` which assembles the condition matrix  $\underline{\underline{M}}_n$  for a given incident angle  $\theta$  and frequency  $\omega$ . In the process, the layer specific wave vector  $\underline{k}_i$  is calculated by `Layer.getK()` for every mode from incident angle and frequency. The phase velocities are accessed from the object. Calculation of polarisation vectors (`getPolarisations()`) and strain tensor (`getStrain()`) follows the same principle. The functions `Intensity()` implementing equation 2.45 and `getPropagationMatrix()`

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implementing  $\underline{P}_n$  from equation 2.57 support the later mentioned methods from `Reflector`.

To construct a layer structure that can be evaluated by the simulation tools of `Reflector`, those `Layer` objects can be assembled in a list, which indicates the order of `Layers` in the reflector structure. Together with the defined thicknesses of each `Layer` object this forms an unambiguous representation of a physical layer structure.

This list of `Layer` objects is then passed to the constructor of a `Reflector` object and stored as deep copy of the initial list to avoid bugs through references to the same object. Besides, the layer thicknesses are extracted and stored internally in a separate list which is used exclusively over the object stored thicknesses. The first and the last given layer are assumed to embed the remaining layers, so that their thickness defined in the `Layer` object has no meaning. Thus the thicknesses of first and last layer are set to 0 in `Reflector.__init__()` for convenience in the simulation functions. The thickness configuration can be changed with `setThicknesses()` and a full text representation of the current configuration can be generated with `info()`.

### 3.1.1 Algorithms and their Scope of Application

There are four methods implemented to calculate the transmittivity function  $\mathcal{T}_i(\omega, \theta)$  differing in numerical stability and speed. Those are `TransferMethod()` and `TransferMethodMP()`, which implement the Transfer Matrix Method and on the other side `LSEMethod()` and `SingleLSEMethod()`, which implement the described LSE method. All of those methods take the same kind of parameters, which are frequency  $\omega$ , incident angle  $\theta$  and the coefficient vector of the initial wave  $\underline{t}_0$ . The returned values are the transmission and reflection coefficients  $\underline{t}_N$  and  $\underline{r}_0$  as well as transmittivity  $\mathcal{T}$  and reflectivity  $\mathcal{R}$ . The last two quantities are evaluated by calling `Reflector.Intensities()` in turn relies on the layer specific intensities from `Layer.Intensity()`.

To explain the coexistence of the methods, it is helpful to describe and analyse them in the order of their development. The challenge here was not to get a working implementation, but an implementation that is able to produce valid results in convenient time for the problem described in the introduction.

One important tool for validating the obtained transmittivity values is the sanity check. Each of these methods is checked for energy conservation by evaluating the validity of  $\mathcal{R} + \mathcal{T} = 1$ , the sum of reflectivity  $\mathcal{R}$  and transmittivity  $\mathcal{T}$  which was discussed in section 2.2.2. It offers a fast test for reasonable results. However,

it can not exclude all kinds of errors so that additional validation against known cases will be provided.

The first attempt was `TransferMethod()`. The TMM offers the most promising properties with regard to computation time for large systems, because existing matrix multiplication algorithms as implemented in the `numpy` package can be used and the addition of layers to the problem only increases the computation time linearly. The latter becomes apparent when considering equation 2.60.

Although the implemented algorithm is fast as expected and is able to pass the sanity check, a significant limitation occurs for layer structures with more than 6 interfaces and frequencies over a few GHz. The sanity of the values of such configurations begins to decay for large angles. Large regions in angle frequency space establish, where the resulting values are covered by a strong noise leading to values of almost diverging magnitude. Examples of that behaviour are shown in the appendix.

It is assumed that this is caused during calculation of very large and very small values with limited precision. This especially occurs in the propagation matrix  $\underline{\underline{P}}_n$ , which is responsible for the propagation of the coefficients through a layer. It contains both large and small values in the case of evanescent waves. The wave vector adopts a negative complex part so that the lower diagonal entries increase exponentially with layer thickness while the upper diagonal entries decrease with layer thickness.

more specific explanation?

The effect of the nature of  $\underline{\underline{P}}$  was investigated by analysing the continuity in  $\theta$  of matrix entries with a debugging tool. The intention was to find the operation that causes the most numerical instability, which shows in non continuous functionality of the result.

However, it was found that the mere composition to the complete transfer matrix  $\underline{\underline{S}}$  is less relevant for that effect than the final calculation of the coefficients in equations 2.62 and 2.63. These are unfortunately essential to the Transfer Matrix Method.

As other explanations over the source of the numerical instabilities could not be verified and after several unsuccessful attempts of improving numerical stability, the lse method was considered. It represents the rather heuristic approach to solve the system of equations in one step with the benefit, that the exponentials with magnifying exponential can be reformulated as attenuating exponential functions. Therefore, the numerical issues could be improved in a manner that the required computations can be undertaken successfully.

However, this method has other disadvantages. Firstly, the time complexity is no longer linear in the number of layers. The method is significantly slower for small problems which makes large problems much more time consuming than

with the TMM. Another limitation is given by the `numpy` focused coding style, where python loops are avoided by initialising arrays and exploiting faster matrix operations. Here the loop over arrays of initial angles  $\theta$  and the related frequencies  $\omega$  was implemented by stacking the matrices and vectors for a single value pair along additional dimensions. As a result, the computation transmittivity values for several hundreds of frequencies with several hundreds of angles each at one time can require more storage than a typical RAM can offer.

Another approach was to use sparse matrices with the package `sparse` which is compatible with `numpy` and implements much more efficient storage of matrices with many zero valued elements. Although the present matrices fulfil this criterion, the assembly of those matrices from `numpy` dense matrices proved too slow in comparison to the gained speed. This is possibly related to the circumstance, that the module wraps the computational methods of `scipy.sparse.linalg` which only uses two-dimensional sparse matrices.

For this reason, the method `SingleLSEMethod()` was created as linearised method using python loops. In each iteration, it passes a single angle and frequency to `LSEMethod()`. For that case it was also optimised by using the sparse system solver `spsolve()` from `scipy.sparse.linalg` which improves the performance further.

As the LSE Method also encounters numerical problems for larger structures a fourth attempt was made by using the module `mpmath`. It provides types with arbitrary numerical precision. A full implementation of the Transfer Matrix Method with `mpmath` is given by `TransferMethodMP()` and several submethods ending with `MP`. This was done by Tobias Hangleiter in analogy to the existing TMM implementation. The `mpmath` approach is able to deliver numerically stable results as expected. However, the computation time is significantly higher than for all other methods, so that it is only used in cases, where the other methods fail.

In addition, another optimisation can be implemented. All these algorithms rely on several subroutines like for example `Layer.getConditionMatrix()`, which calculates the matrix  $\underline{\underline{M}}_n$  from equation 2.59. In this and other cases, the function is called several times with the same arguments due to the fact, that the propagating angles in each Layer are equal in each Layer of the same type. For this reason, the results of these subroutines are cached. As conventional caching decorators require the function parameters to be hashable and this does not apply for `numpy` arrays, caching was implemented by hand. The essential part of this are the hash function `Hash()` which converts an arbitrary number of parameters to a hashable tuple. This hash is then used as key in a dictionary for the results of the hashed function. For each cached function there is such a dictionary stored in either `Reflector` or `Layer`.



### 3.1.2 Plots and Scripts for Evaluation

In order to evaluate the total heat transfer, these methods are used to calculate an array of transmittivities over all angles  $\theta \in [0^\circ, 90^\circ]$  and over all frequencies  $\omega \in (0\text{s}^{-1}, 30 \cdot 10^{11}\text{s}^{-1}]$ . The integral 1.1 is then evaluated first by integrating over the angles which is done by `Reflector.spectralTransmission()` using the method `numpy.trapz()`. The weighting factor  $\cos\theta$  is also considered so that the resulting values represent spectral transmittivity in z-direction. The total heat flow is calculated with `transmittedPower()` by evaluating the described integrant at the same frequencies as the spectral transmittivity and integrating it analogously to the angles.

mention  $\Delta P$

The intermediate results of this calculation may be of interest as well. For this purpose, `plotfunctions.py` provides a number of plot functions and scripts that were also used to generate the plots in the later course of this thesis. The first important plot function is `plotAngularTransmission()`. It visualizes the reflection and transmission coefficients of all three modes as well as the resulting transmittivity and reflectivity as functions of incident angle for a fixed frequency. This can be viewed in figure .

Angular Plot  
reference

In comparison to that, `plotOverview()` accepts transmittivity values from a region in angle-frequency space as two-dimensional array and is able to provide a three-part plot as in figure . Here the upper left field represents a color plot of the sanity of the transmittivity over frequency and angle axis. Dots coded as 1 denote that the transmittivity at the corresponding angle and frequency was calculated correctly. At the right, the transmittivity values are represented directly in a similar plot. Those two plots help to understand the regions of numerical instabilities but also the behaviour of transparent regions of the reflector under angle or frequency variation. The lower plot, the spectral radiance transmitted through the reflector in comparison to the incident radiance. A textbox also shows the results of the integration over those curves and the ratio of transmitted power. For each plot a boolean parameter is passed to the function to enable or disable the appearance of each plot in the output.

overview plot  
reference

These plot functions also exist in form of a script like function, that generates the necessary transmission values by itself and directly provide the output as plot.

In addition to these evaluation plots, there an additional utility function called `EffectivityOverThickness()`. It is an attempt to automatise the optimisation process that will be described later. The method calculates the ratio of transmitted power as in `transmittedPower()` for a reflector with two different materials with repeating thicknesses. The output is a matrix, where thickness of layer 1 varies along the first axis and the thickness of layer 2 varies along the



second axis. This matrix is again represented as colorplot. This plot was not used here as it is very time consuming to compute and has only limited significance.

Example  
Workflow  
still advan-  
tageous?

SIMULATIONS

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## CONCLUSION

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## APPENDIX

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### A.1 CONCRETE FORM OF THE CONDITION MATRICES

### A.2 DETAILED PLOTS PICTURING SCOPES OF NUMERICAL INSTABILITES