
Modeling and Control of Hybrid Systems

Assignment report - Group A 37

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Introduction

This report considers modeling problems that are tackled with a hybrid system approach. First of all, a simple hybrid system is modeled and described in order to get acquainted with the idea of hybrid modeling. Next, an Adaptive Cruise Control (ACC) problem is considered. The aim is to analyse the problem and in the end find an optimal control input for which the car follows a given speed profile. This is done by transforming the problem into a Mixed Logical Dynamical (MLD) model and apply a Model Predictive Controller (MPC) on it.

1 Hybrid System example

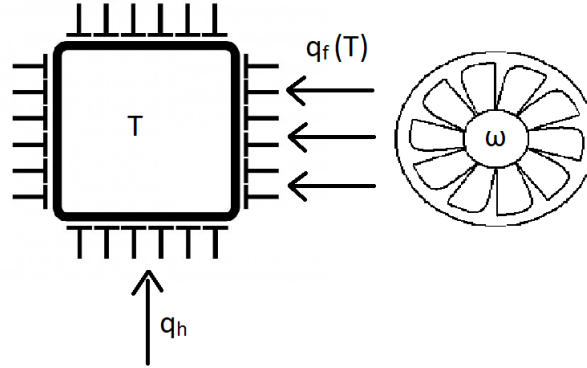


Figure 1: CPU cooler system

First, for a hybrid modeling exercise, a simple CPU cooling system is considered and shown in figure 1. On the left side, the CPU is shown that has a perfectly measurable temperature and a constant heat energy input as an effect of electrical power dissipation. The fan is operational in three modes, e.g. three fan speeds, that induce different cooling rates. The parameters are denoted in table 1. These help understand the system dynamics as given in equations 1 and 2, which describe the CPU temperature evolution and the depending CPU mode respectively. Note that the first equation includes a minus sign in front of the fan contribution, as the heat added is negative.

Parameter	interpretation
T	CPU Temperature
q_h	heat CPU input
$q_f(T)$	heat input due to fan convection
ω	fan speed

Table 1: System Parameters

$$\dot{T} = q_h - q_f \quad (1)$$

$$q_f(T) = \begin{cases} q_0 & \text{if } T < T_0 \\ q_{c1} & \text{if } T_0 \leq T < T_1 \\ q_{c2} & \text{if } T_1 \leq T \end{cases} \quad (2)$$

Next, a hybrid automaton of the system can be constructed. It is given by the 8-tuple defined in equation (3) and elaborated in equation (4). The physical representation is shown in figure 2.

$$H = \{Q, X, f, Init, Inv, E, G, R\} \quad (3)$$

$$\begin{aligned}
Q &= \{S_1, S_2, S_3\} \\
X &= \mathbb{R} \\
f : Q \times X &\rightarrow X \\
&: \{(S_1, T) \rightarrow q_h, (S_2, T) \rightarrow q_h - q_{c1}, (S_3, T) \rightarrow q_h - q_{c2}\} \\
Init &= Q \times X \\
Inv &= \{(S_1, T < T_0), (S_2, T \in [T_0, T_1]), (S_3, T > T_1)\} \\
E &= \{(S_1, S_2), (S_2, S_1), (S_2, S_3), (S_3, S_2)\} \\
G &= \{((S_1, S_2), T > T_0), ((S_2, S_1), T \leq T_0), ((S_2, S_3), T > T_1), ((S_3, S_2), T \leq T_1)\} \\
R : E &\rightarrow P(X \times X) \\
&: E \rightarrow (T^- = T^+)
\end{aligned} \tag{4}$$

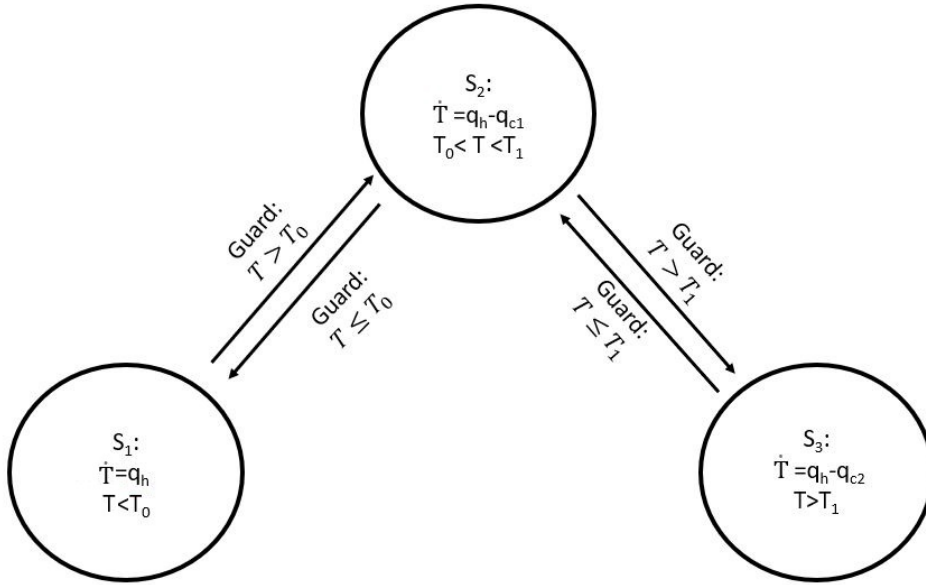


Figure 2: Hybrid Automaton

2 Adaptive cruise control

2.1 Control magnitudes and extreme velocities

Next, the ACC car is considered. First, using equation 5, a number of quantities are demanded.

$$\frac{d}{dt} \begin{bmatrix} x(t) \\ v(t) \end{bmatrix} = \begin{bmatrix} v(t) \\ \frac{1}{m} \left(\frac{bu(t)}{1+\gamma g(t)} - cv^2(t) \right) \end{bmatrix} \quad (5)$$

The maximum velocity is obtained when the maximum drive force equals the air friction force. So that

$$\frac{1}{m} \left(\frac{bu_{max}}{1+\gamma g_{max}} - cv_{max}^2(t) \right) = 0. \quad (6)$$

Which gives

$$v_{max} = \left(\frac{bu_{max}}{c(1+\gamma g_{max})} \right)^{\frac{1}{2}}. \quad (7)$$

If equation 5 is considered again, it can be seen that it consists of a positive and negative part. The maximum acceleration should occur when the positive part is at its largest and the negative part is at its smallest. This implies: $u = u_{max}$ and $v = 0$ in this case and yields:

$$a_{max} = \frac{1}{m} \frac{bu_{max}}{1+\gamma g(t)} \quad (8)$$

The same reasoning can be used for finding the maximal deceleration. Now the first term has to be minimal and the second term has to be maximal. This will happen at the maximal velocity when suddenly the maximal brake input is applied. This implies:

$$a_{min} = \frac{1}{m} \left(\frac{bu_{min}}{1+\gamma g(t)} - cv_{max}^2(t) \right) \quad (9)$$

The found values are denoted in table 2. Besides, figure 3 describes a simulation run of the car with all the prescribed dynamics. It can be seen that indeed the arguments with respect to the extremes hold.

Table 2: Model quantities

Entity	Value
Maximum speed	$57.72ms^{-1}$
Maximum acceleration	$3.22ms^{-2}$
Maximum deceleration	$3.33ms^{-2}$

2.2 Friction approximation

Now the task is to construct a PWA that approximates the quadratic friction curve using two regions. This can be done by minimizing the integral

$$\int_0^{v_{max}} (V(v) - P(v))^2 dv \quad (10)$$

The two line segments are given as follows:

- The first one goes through the origin and through point (α, β)
- The second one goes through point (α, β) and through point (v_{max}, cv_{max}^2)

Thus

$$P(v) = P_1(v) + P_2(v) \quad (11a)$$

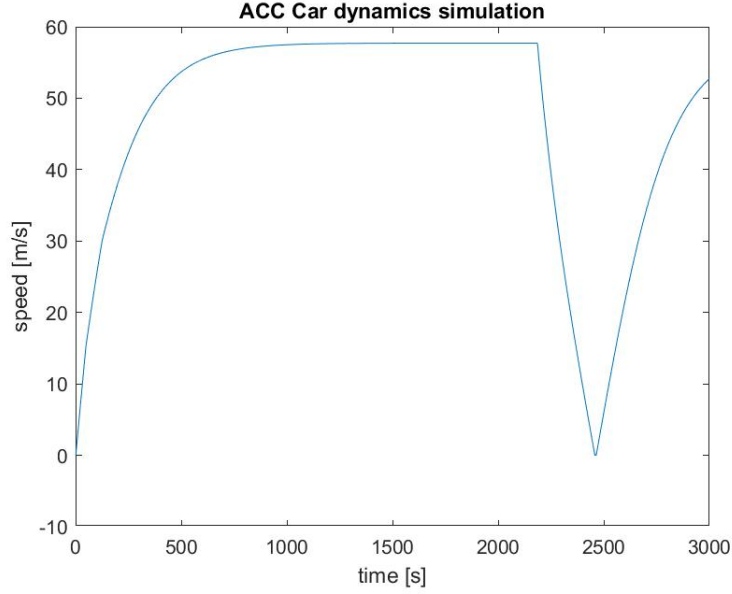


Figure 3: Simulation of the ACC Car

$$P_1(v) \begin{cases} \frac{\beta}{\alpha} v & v < \alpha \\ 0 & v \geq \alpha \end{cases} \quad (11b)$$

$$P_2(v) \begin{cases} 0 & v < \alpha \\ \frac{cv_{max}^2 - \beta}{v_{max} - \alpha} v + \beta - \frac{cv_{max}^2 - \beta}{v_{max} - \alpha} \alpha & v \geq \alpha \end{cases} \quad (11c)$$

The Equation 10 can then be divided into two sections as

$$A = \int_0^\alpha (V(v) - P_1(v))^2 dv + \int_\alpha^{v_{max}} (V(v) - P_2(v))^2 dv. \quad (12)$$

Using the "Symbolic Math Toolbox" provided in MatLab, a function $A(\alpha, \beta)$ is determined. To find the extremes of A the system of equations

$$\begin{cases} \frac{dA}{d\alpha} = 0 \\ \frac{dA}{d\beta} = 0 \end{cases} \quad (13)$$

is solved with MatLab's solve function. This yields three pairs of (α, β) , of which only one is valid. That is, $\alpha \in [0, v_{max}]$ and $\beta \in [0, cv_{max}^2]$. When evaluating the second derivative of A with respect to both α and β , both turn out to be greater than 0. Thus, this extreme is a minimum. The values are

$$\begin{cases} \alpha \approx 28.8575 \\ \beta \approx 249.8269. \end{cases} \quad (14)$$

2.3 Approximation compared to reality

Right now, there is looked upon the differences between the different friction force models. The friction dynamics correspond with those presented in figure 4. The PWA-model friction dynamics are shown by the red line segments, the true friction force by the blue curve. The velocities at which the different model coincide are called $v_{cross,1}$ and $v_{cross,2}$, which are about $21.5 \frac{m}{s}$ and $36.5 \frac{m}{s}$ respectively. This shows that the PWA-modelled friction force is in reality a bit lower for all velocities $v < v_{cross,1}$ and $v > v_{cross,2}$ and higher for all velocities $v_{cross,1} < v < v_{cross,2}$.

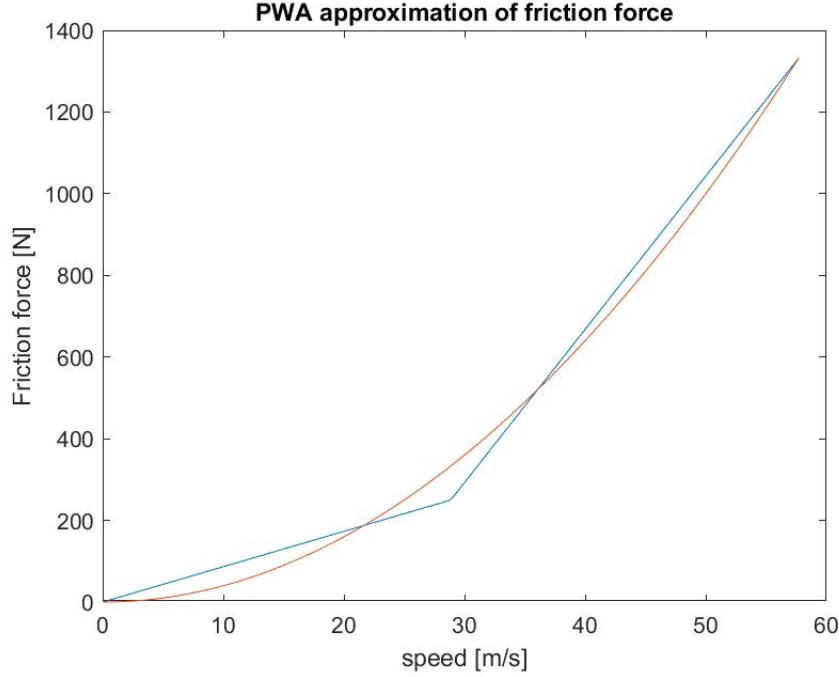


Figure 4: Different friction models

This means that the model will build up an error that correspond with time integral of the difference in the 2 curves. Namely, in the time the velocity is lower than $v_{cross,1}$ and larger than $v_{cross,2}$, the PWA model will introduce an acceleration larger than in reality, in the time the velocity is between $v_{cross,1}$ and $v_{cross,2}$, the PWA model will introduce an acceleration smaller than reality. If a sinusoidal throttle input is applied, as shown in figure 5a, it can be seen that the properties of the input make the error cancel out until some extend, as the time spend in both speed regions is more or less equal as can be seen in figure 5b.

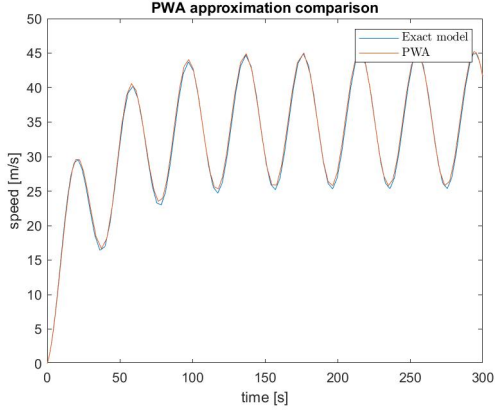
2.4 PWA model

Considering a fully functional gear system again, the piecewise affine (PWA) system model with friction approximation, can be created with the found results. Since there are two regions in the friction approximation, and the gear creates three regions in the driving force, the system will have six regions. These regions are denoted as $\mathbb{X}_i, i \in \{1, 2, 3, 4, 5, 6\}$ and the corresponding dynamics as f_i . Defining the regions as

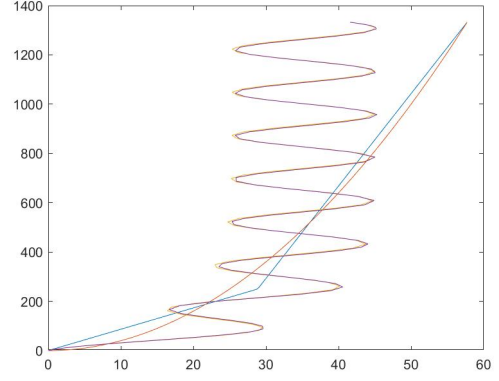
$$\begin{aligned} \mathbb{X}_1 &= \{(x, v) \mid v < v_{12}, v < \alpha\} & \mathbb{X}_4 &= \{(x, v) \mid v < v_{12}, v \geq \alpha\} \\ \mathbb{X}_2 &= \{(x, v) \mid v \geq v_{12}, v < v_{23}, v < \alpha\} & \mathbb{X}_5 &= \{(x, v) \mid v \geq v_{12}, v < v_{23}, v \geq \alpha\} \\ \mathbb{X}_3 &= \{(x, v) \mid v \geq v_{23}, v < \alpha\} & \mathbb{X}_6 &= \{(x, v) \mid v \geq v_{23}, v \geq \alpha\} \end{aligned} \quad (15)$$

creates a division of the whole of \mathbb{R}^2 . However, since α is found to be between v_{12} and v_{23} , \mathbb{X}_3 and \mathbb{X}_4 are unreachable, so no dynamics will be defined for these regions. Lastly, three driving force constants are defined as

$$F_{drive,i} = \frac{b}{1 + \gamma i}, \quad i \in \{1, 2, 3\}. \quad (16)$$



(a) Model input and exact input



(b) Input magnitude analysis

Figure 5: Throttle input properties

Then the PWA model with state $x = (x, v)$ is

$$\frac{d}{dt}x = \begin{cases} \begin{bmatrix} x_2 \\ \frac{1}{m} (F_{\text{drive},1}u - P_1(x_2)) \end{bmatrix} & , x \in \mathbb{X}_1 \\ \begin{bmatrix} x_2 \\ \frac{1}{m} (F_{\text{drive},2}u - P_1(x_2)) \end{bmatrix} & , x \in \mathbb{X}_2 \\ \begin{bmatrix} x_2 \\ \frac{1}{m} (F_{\text{drive},2}u - P_2(x_2)) \end{bmatrix} & , x \in \mathbb{X}_5 \\ \begin{bmatrix} x_2 \\ \frac{1}{m} (F_{\text{drive},3}u - P_2(x_2)) \end{bmatrix} & , x \in \mathbb{X}_6 \end{cases} \quad (17)$$

2.5 Discretization

In this step, the PWA model is discretized using forward euler discretization. The discrete system dynamics then become

$$x(k+1) = x(k) + \dot{x}\Delta t. \quad (18)$$

The results can be shown in the appendix at code section 2.5.

2.6 MLD model

From here on only the second state of the system is considered. The goal of this section is to cast the dynamics into the Mixed Logical Dynamical (MLD) system framework. First, suited expressions for the drag force and driving force will be determined, after which they can be expressed in the MLD format. For the MLD format it is important to remember the ranges in which our state and input live, namely $x \in [0, v_{\max}]$ and $u \in [u_{\min}, u_{\max}]$.

The drag force is expressed as

$$P(x) = \begin{cases} kx & x < \alpha \\ qx + r & x \geq \alpha \end{cases} \quad (19)$$

in the PWA model, where

$$\begin{aligned} k &= \frac{\beta}{\alpha} \\ q &= \frac{cv_{max}^2 - \beta}{v_{max} - \alpha} \\ r &= \beta - \frac{cv_{max}^2 - \beta}{v_{max} - \alpha} \alpha. \end{aligned} \quad (20)$$

We introduce a boolean variable δ_P such that $[x \geq \alpha] \iff [-x + \alpha \leq 0] \iff [\delta_P = 1]$. This leads to the constraints for δ_P to be

$$\begin{aligned} -x + \alpha &\leq \alpha(1 - \delta_P) \\ -x + \alpha &\geq \epsilon + (-v_{max} + \alpha - \epsilon)\delta_P, \end{aligned} \quad (21)$$

where ϵ is machine precision. The PWA function for the drag force can then be expressed as

$$\begin{aligned} P(x) &= kx + \delta_P(-kx + qx + r) = (q - k)\delta_P x + kx + \delta_P r \\ &= (q - k)z_P + kx + \delta_P r, \end{aligned} \quad (22)$$

where $z_P = \delta_P x$, or equivalently

$$\begin{aligned} z_P &\leq v_{max}\delta_P \\ z_P &\geq 0 \\ z_P &\leq x \\ z_P &\geq x - v_{max}(1 - \delta_P). \end{aligned} \quad (23)$$

The driving force in the PWA model is expressed as

$$F_{\text{drive},i}u = \frac{b}{1 + \gamma i}u, \quad i \in \{1, 2, 3\}. \quad (24)$$

To rewrite this expression, two more boolean variables are introduced. The first is δ_2 such that $[x \geq v_{12}] \iff [-x + v_{12} \leq 0] \iff [\delta_2 = 1]$. The second is δ_3 such that $[x \geq v_{23}] \iff [-x + v_{23} \leq 0] \iff [\delta_3 = 1]$. Thus the associated constraints for these variables are

$$\begin{aligned} -x + v_{12} &\leq v_{12}(1 - \delta_2) \\ -x + v_{12} &\geq \epsilon + (-v_{max} + v_{12} - \epsilon)\delta_2, \end{aligned} \quad (25)$$

and

$$\begin{aligned} -x + v_{23} &\leq v_{23}(1 - \delta_3) \\ -x + v_{23} &\geq \epsilon + (-v_{max} + v_{23} - \epsilon)\delta_3. \end{aligned} \quad (26)$$

It then becomes possible to rewrite the driving force into

$$F_{\text{drive}}u = F_{\text{drive},1}u + \delta_2u(F_{\text{drive},2} - F_{\text{drive},1}) + \delta_3u(F_{\text{drive},3} - F_{\text{drive},2}). \quad (27)$$

Again two auxiliary variables, z_2 and z_3 , are introduced as $z_2 = \delta_2u$ and $z_3 = \delta_3u$ under the constraints

$$\begin{aligned} z_2 &\leq u_{max}\delta_2 \\ z_2 &\geq u_{max}\delta_2 \\ z_2 &\leq u - u_{min}(1 - \delta_2) \\ z_2 &\geq u - u_{max}(1 - \delta_2) \end{aligned} \quad (28)$$

and

$$\begin{aligned} z_3 &\leq u_{max}\delta_3 \\ z_3 &\geq u_{max}\delta_3 \\ z_3 &\leq u - u_{min}(1 - \delta_3) \\ z_3 &\geq u - u_{max}(1 - \delta_3). \end{aligned} \quad (29)$$

Then

$$F_{\text{drive}}u = F_{\text{drive},1}u + z_2(F_{\text{drive},2} - F_{\text{drive},1}) + z_3(F_{\text{drive},3} - F_{\text{drive},2}). \quad (30)$$

All of the equations are now ready to be written in the MLD format as

$$\begin{aligned} x(k+1) &= Ax(k) + B_1u(k) + B_2\delta(k) + B_3z(k) \\ E_1x(k) + E_2u(k) + E_3\delta(k) + E_4z(k) &\leq g_5. \end{aligned} \quad (31)$$

Starting with the dynamics, substituting the new results into Equation 18 yields

$$\begin{aligned} x(k+1) &= x(k) + \dot{x}\Delta t = x(k) + \frac{\Delta t}{m} (F_{\text{drive}}u(k) - P(x(k))) \\ &= x(k) + \frac{\Delta t}{m} (F_{\text{drive},1}u(k) + z_2(k)(F_{\text{drive},2} - F_{\text{drive},1}) + z_3(k)(F_{\text{drive},3} - F_{\text{drive},2}) \\ &\quad - [(q-k)z_P(k) + kx(k) + \delta_P(k)r]) \\ &= x(k) + \frac{\Delta t}{m} F_{\text{drive},1}u(k) + \frac{\Delta t}{m} (F_{\text{drive},2} - F_{\text{drive},1})z_2(k) + \frac{\Delta t}{m} (F_{\text{drive},3} - F_{\text{drive},2})z_3(k) \\ &\quad + \frac{\Delta t}{m} [(k-q)z_P(k) - kx(k) - r\delta_P(k)] \\ &= x(k) + \frac{\Delta t}{m} F_{\text{drive},1}u(k) + \frac{\Delta t}{m} (F_{\text{drive},2} - F_{\text{drive},1})z_2(k) + \frac{\Delta t}{m} (F_{\text{drive},3} - F_{\text{drive},2})z_3(k) \\ &\quad + \frac{\Delta t}{m} (k-q)z_P(k) - \frac{\Delta t}{m} kx(k) - \frac{\Delta t}{m} r\delta_P(k) \\ &= \underbrace{\left(1 - \frac{\Delta t}{m}k\right)x(k)}_A + \underbrace{\frac{\Delta t}{m}F_{\text{drive},1}u(k)}_{B_1} + \underbrace{\begin{pmatrix} 0 & 0 & -\frac{\Delta t}{m}r \end{pmatrix}}_{B_2} \begin{bmatrix} \delta_2(k) \\ \delta_3(k) \\ \delta_P(k) \end{bmatrix} \\ &\quad + \underbrace{\frac{\Delta t}{m} \begin{pmatrix} (F_{\text{drive},2} - F_{\text{drive},1}) & (F_{\text{drive},3} - F_{\text{drive},2}) & (k-q) \end{pmatrix}}_{B_3} \begin{bmatrix} z_2(k) \\ z_3(k) \\ z_P(k) \end{bmatrix}. \end{aligned} \quad (32)$$

All of the mentioned constraints can be written as

$$\begin{aligned}
 & \underbrace{\begin{bmatrix} 1 \\ -1 \\ 0 \\ 0 \\ -1 \\ 1 \\ 0 \\ 0 \\ 0 \\ -1 \\ 1 \\ -1 \\ 1 \\ 0 \\ 0 \\ 0 \\ 0 \\ -1 \\ 1 \\ 0 \\ 0 \\ 0 \\ 0 \end{bmatrix}}_{E_1} x(k) + \underbrace{\begin{bmatrix} 0 \\ 0 \\ 1 \\ -1 \\ 0 \\ 0 \\ 0 \\ 0 \\ 0 \\ 0 \\ 0 \\ 0 \\ 0 \\ -1 \\ 1 \\ 0 \\ 0 \\ -1 \\ 1 \\ 0 \\ 0 \\ -1 \\ 1 \end{bmatrix}}_{E_2} u(k) + \underbrace{\begin{bmatrix} 0 & 0 & 0 \\ 0 & 0 & 0 \\ 0 & 0 & 0 \\ 0 & 0 & 0 \\ 0 & 0 & 0 \\ 0 & 0 & \alpha \\ 0 & 0 & (-v_{max} + \alpha - \epsilon) \\ 0 & 0 & -v_{max} \\ 0 & 0 & 0 \\ 0 & 0 & 0 \\ 0 & 0 & v_{max} \\ v_{12} & 0 & 0 \\ (-v_{max} + v_{12} - \epsilon) & 0 & 0 \\ -u_{max} & 0 & 0 \\ u_{min} & 0 & 0 \\ -u_{min} & 0 & 0 \\ u_{max} & 0 & 0 \\ 0 & v_{23} & 0 \\ 0 & (-v_{max} + v_{23} - \epsilon) & 0 \\ 0 & -u_{max} & 0 \\ 0 & u_{min} & 0 \\ 0 & -u_{min} & 0 \\ 0 & u_{max} & 0 \end{bmatrix}}_{E_3} \begin{bmatrix} \delta_2(k) \\ \delta_3(k) \\ \delta_P(k) \end{bmatrix} \\
 & + \underbrace{\begin{bmatrix} 0 & 0 & 0 \\ 0 & 0 & 0 \\ 0 & 0 & 0 \\ 0 & 0 & 0 \\ 0 & 0 & 0 \\ 0 & 0 & 0 \\ 0 & 0 & 1 \\ 0 & 0 & -1 \\ 0 & 0 & 1 \\ 0 & 0 & -1 \\ 0 & 0 & 0 \\ 0 & 0 & 0 \\ 1 & 0 & 0 \\ -1 & 0 & 0 \\ 1 & 0 & 0 \\ -1 & 0 & 0 \\ 0 & 0 & 0 \\ 0 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & -1 & 0 \\ 0 & 1 & 0 \\ 0 & -1 & 0 \end{bmatrix}}_{E_4} \begin{bmatrix} z_2(k) \\ z_3(k) \\ z_P(k) \end{bmatrix} \leq \underbrace{\begin{bmatrix} v_{max} \\ 0 \\ u_{max} \\ -u_{min} \\ 0 \\ \alpha - \epsilon \\ 0 \\ 0 \\ 0 \\ v_{max} \\ 0 \\ v_{12} - \epsilon \\ 0 \\ 0 \\ -u_{min} \\ u_{max} \\ 0 \\ v_{23} - \epsilon \\ 0 \\ 0 \\ -u_{min} \\ u_{max} \end{bmatrix}}_{g_5}
 \end{aligned} \tag{33}$$

When the PWA and MLD model are compared for different speeds and inputs, they are largely equal, except for two points. This is demonstrated in Figure 6. At exactly v_{12} and v_{23} the models differ and it is assumed that this occurs because the one model chooses a different gear than the other at these speeds. However, in practise the chances of the state being exactly v_{12} or v_{23} is extremely small, and even then the relative error to the velocities is small.

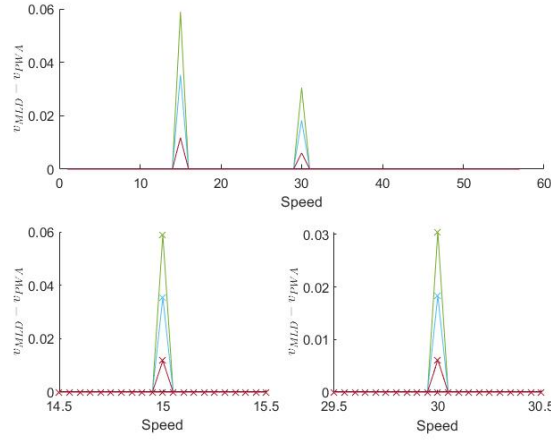


Figure 6: Comparison of a one step state evolution of the PWA and MLD model

2.7 Implicit MPC formulation

This section describes the creation of an MPC controller that optimises speed tracking for the MLD model. The controller considers a prediction horizon N_p and a control horizon $N_c < N_p$ after which the input will remain constant. The state for a time k is defined as

$$x(k) = [v(k) \quad u(k) \quad \delta_2(k) \quad \delta_3(k) \quad \delta_P(k) \quad z_2(k) \quad z_3(k) \quad z_P(k)]^T. \quad (34)$$

The optimisation parameter then becomes a vertical concatenation of $x(k)$ through $x(k + N_p)$.

2.7.1 The cost function

The used optimisation function is the `glpk` function from the "MPT" toolbox that optimises a vector x for cost Cx such that $\Omega x \sim \omega$, where \sim can be one of a number of relations, defined per row of Ω . However in this report only equality and upper bound constraints are considered. The cost function to be optimised is

$$\min_{\mathbf{u}} J(x) = \|\mathbf{v} - \mathbf{v}_{\text{ref}}\|_1 + \lambda \|\mathbf{u}\|_1, \quad (35)$$

where $\lambda \in \mathbb{R}^+$. To write this in linear form we introduce two auxiliary variable vectors of length N_p , ρ_x and ρ_u for the state and input norm respectively. An equivalent optimisation problem is then

$$\begin{aligned} \min_{\mathbf{u}} J(x) &= \mathbf{1}^{1 \times N_p} \rho_x + \lambda \mathbf{1}^{1 \times N_p} \rho_u \\ \text{s.t.} \quad &\mathbf{v} - \mathbf{v}_{\text{ref}} \leq \rho_x \\ &\mathbf{v} - \mathbf{v}_{\text{ref}} \geq -\rho_x \\ &\mathbf{u} \leq \rho_u \\ &\mathbf{u} \geq -\rho_u. \end{aligned} \quad (36)$$

These constraints are also equivalent to

$$\begin{aligned} &\mathbf{v} - \rho_x \leq \mathbf{v}_{\text{ref}} \\ &-\mathbf{v} - \rho_x \leq -\mathbf{v}_{\text{ref}} \\ &\mathbf{u} - \rho_u \leq 0 \\ &-\mathbf{u} - \rho_u \leq 0. \end{aligned} \quad (37)$$

The vectors ρ_x and ρ_u are appended to the optimisation parameter, which is now complete. The total optimisation parameter is of size $8N_p + 2N_p = 10N_p$ and is of the form

$$x(k) = \begin{bmatrix} v(k) & u(k) & \delta_2(k) & \delta_3(k) & \delta_P(k) & z_2(k) & z_3(k) & z_P(k) \\ \vdots & \vdots & \vdots & \vdots & \vdots & \vdots & \vdots & \vdots \\ v(k+N_p) & u(k+N_p) & \delta_2(k+N_p) & \delta_3(k+N_p) & \delta_P(k+N_p) & z_2(k+N_p) & z_3(k+N_p) & z_P(k+N_p) \\ \rho_x \\ \rho_u \end{bmatrix}^T \quad (38)$$

The cost C then becomes

$$C = \begin{bmatrix} \mathbb{0}^{1 \times 8N_p} & \mathbf{1}^{1 \times N_p} & \lambda \mathbf{1}^{1 \times N_p} \end{bmatrix} \quad (39)$$

2.7.2 Auxiliary optimisation constraints in matrix form

To pass the constraints in Equation 37, they have to be written in matrix form $Mx \leq b_\rho$. To extract \mathbf{v} from x , the matrix X is defined as

$$X = \begin{bmatrix} I_{N_p} \otimes [1 & \mathbb{0}^{1 \times 7}] & \mathbb{0}^{N_p \times 2N_p} \end{bmatrix}, \quad (40)$$

where \otimes is the Kronecker product. Likewise to extract \mathbf{u} from x ,

$$U = \begin{bmatrix} I_{N_p} \otimes [0 & 1 & \mathbb{0}^{1 \times 6}] & \mathbb{0}^{N_p \times 2N_p} \end{bmatrix}. \quad (41)$$

To extract ρ_x and ρ_u , the matrices Γ_x and Γ_u are defined similarly as

$$\Gamma_x = \begin{bmatrix} \mathbb{0}^{N_p \times 8N_p} & I_{N_p} & \mathbb{0}^{N_p \times N_p} \end{bmatrix} \quad (42)$$

and

$$\Gamma_u = \begin{bmatrix} \mathbb{0}^{N_p \times 9N_p} & I_{N_p} \end{bmatrix}. \quad (43)$$

Then the constraints can be written as

$$\underbrace{\begin{bmatrix} X - \Gamma_x \\ -X - \Gamma_x \\ U - \Gamma_u \\ -U - \Gamma_u \end{bmatrix}}_M x \leq \underbrace{\begin{bmatrix} \mathbf{v}_{\text{ref}} \\ -\mathbf{v}_{\text{ref}} \\ \mathbb{0}^{N_p \times 1} \\ \mathbb{0}^{N_p \times 1} \end{bmatrix}}_{b_\rho}. \quad (44)$$

2.7.3 System constraints

The system of course incurs its own constraints as well, these are the dynamics constraint, the logical constraint, and the comfortability constraint. The dynamics constraint is of the form

$$v(k+1) = Hx \iff v(k+1) - Hx = 0, \quad (45)$$

where $H = \begin{bmatrix} A & B_1 & B_2 & B_3 & \mathbb{0}^{1 \times 2N_p} \end{bmatrix}$. Let us introduce a notation for off diagonal identity matrices, ${}_n I_m$, such that the n th off diagonal are ones, with towards the upper right being positive, and the matrix is of size $m \times m$. For example

$${}_1 I_3 = \begin{bmatrix} 0 & 1 & 0 \\ 0 & 0 & 1 \\ 0 & 0 & 0 \end{bmatrix}. \quad (46)$$

The extraction matrix, Q , of $v(k+1)$ can then concisely be written as

$$Q = {}_1 I_{N_p} \otimes [1 \quad \mathbb{0}^{1 \times 7}]. \quad (47)$$

The prediction matrix, T , is expressed as

$$T = I_{N_p} \otimes H. \quad (48)$$

Now subtracting these matrices yields the wanted constraint matrix $R = T - Q$, but the last row is an invalid constraint since there is no $x(k + N_p + 1)$ in the state vector to compare with, so it is removed. The complete dynamics constraint is then expressed as

$$Rx = \mathbb{0}. \quad (49)$$

Since the state at every timestep has to fulfill the state constraints, this constraint is easily expressed as

$$(I_{N_p} \otimes E)x \leq (\mathbf{1}^{N_p \times 1} \otimes g_5), \quad (50)$$

where $E = \begin{bmatrix} E_1 & E_2 & E_3 & E_4 & \mathbb{0}^{N_p \times 2N_p} \end{bmatrix}$.

Lastly, this section also introduces the comfortability constraint on the acceleration. Whereas in Equation 8 the maximum acceleration was determined through $u_{max} \rightarrow a_{max}$ in continuous time, now we perform $a_{\text{comf, max}} \rightarrow u_{\text{comf, max}}$ in discrete time. This comes down to taking the discrete acceleration and bounding this, thus

$$-a_{\text{comf, max}} \leq \frac{x(k+1) - x(k)}{\Delta t} \leq a_{\text{comf, max}}. \quad (51)$$

Equivalently written using already defined matrices, this constraint becomes

$$\begin{bmatrix} \frac{-1}{\Delta t}(Q - X) \\ \frac{1}{\Delta t}(Q - X) \end{bmatrix} x \leq \mathbf{1} a_{\text{comf, max}}, \quad (52)$$

where the last row of $Q - X$ is again deleted, because it yields an invalid constraint.

2.7.4 MPC constraints

The implementation of MPC induces a number of extra constraints. These consist of the initialisation constraint and the control horizon constraint. The initialisation constraint equates the first state in the optimisation parameter to equal the initial state, v_0 , or in matrix notation

$$\begin{bmatrix} 1 & \mathbb{0} \end{bmatrix} x = v_0. \quad (53)$$

The control horizon constraint is formulated as the inputs being equal after N_c timesteps, or equivalently

$$\begin{bmatrix} \mathbb{0}^{N_c \times N_p - N_c} & (I_{(N_p - N_c)} - \mathbf{1} I_{(N_p - N_c)}) \otimes [0 \quad 1 \quad \mathbb{0}^{1 \times 6}] \end{bmatrix} x = \mathbb{0}. \quad (54)$$

It is not necessary to define a terminal set for the MPC controller, since the system is stable by nature, and the space in which $v(k)$ lives is control invariant by the definition of v_{max} . Thus, the controller can not destabilise the system, and thus is the controlled system stable for any input.

2.7.5 The controller

The resultant controller takes an initial state, and predicts the future based on that with the corresponding optimal control. An example of this is depicted in Figure 7.

2.8 Closed loop MPC

The MPC control loop is now closed, and the controller will work in feedback with the plant. Now at each time step the prediction is made with the MLD model and the first input determined with that is given to the plant that evolves with the real dynamics. This introduces differences between the prediction and reality, that continuously needs to be corrected by the controller. An example with a constant reference is shown in Figure 8. The oscillations around the reference occur due to the mentioned modeling error.

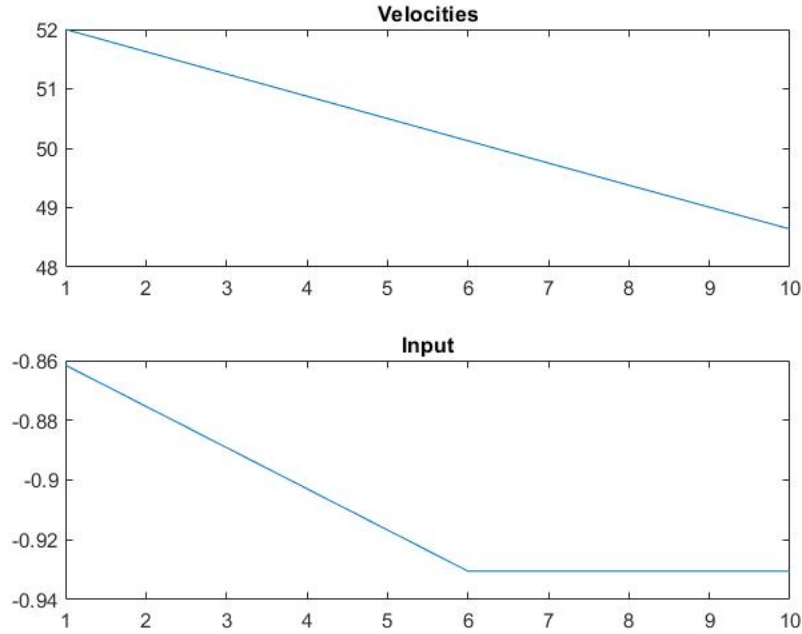


Figure 7: One prediction made by the controller with the expected state and corresponding optimal input; Here $v_0 = 52\text{m/s}$, $v_{ref} = 20\text{m/s}$, $\lambda = 0.3$, $N_p = 10$ and $N_c = 7$

2.9 Time varying reference

A time varying reference is now considered. This can be interpreted as the leading car communicating its speed and future speed to the following car. Two cases are considered, one with a shorter prediction horizon and a longer one. All of the other parameters will be constant and $\lambda = 0.1$.

2.9.1 Short prediction horizon

A pair $(N_p, N_c) = (5, 4)$ is considered. The phase plot and position evolution are shown in Figure 9. The velocity evolution and acceleration is shown in Figure 10. The velocity residual, input and change in input is shown in Figure 11.

2.9.2 Long prediction horizon

Another pair $(N_p, N_c) = (10, 9)$ is considered. The phase plot and position evolution are shown in Figure 12. The velocity evolution and acceleration is shown in Figure 13. The velocity residual, input and change in input is shown in Figure 14.

2.9.3 Comparison

A qualitative description of the differences between the longer and shorter horizon is that the longer horizon acts more conservative than the shorter one. This is most evident in the peak of the velocity evolutions. Besides from this the differences are small, both controllers track the reference quite good, they both change their inputs a lot, they both show similar residual magnitudes, and they both achieve the comfort criterion.

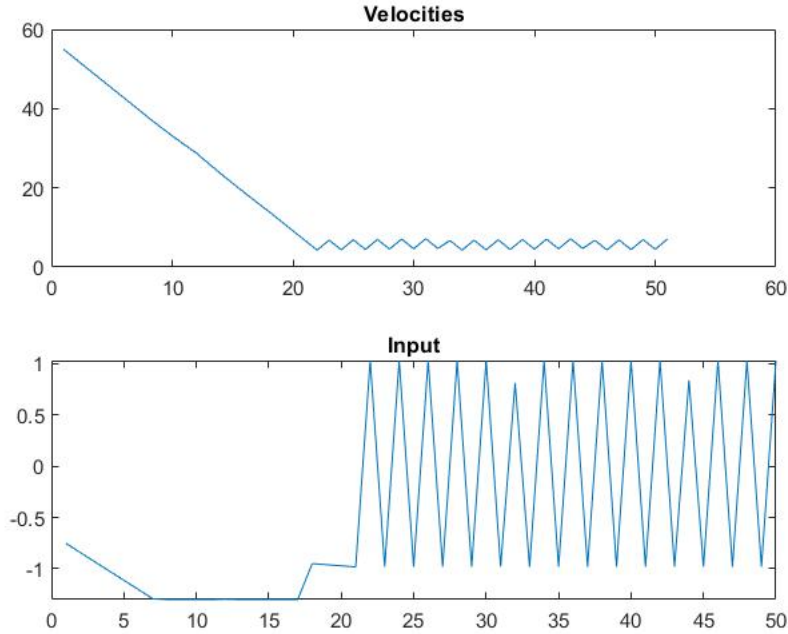


Figure 8: Closed loop control with the state and corresponding optimal input; The horizontal axis is the timestep; Here $v_0 = 55\text{m/s}$, $v_{ref} = 5\text{m/s}$, $\lambda = 0.3$, $N_p = 10$ and $N_c = 7$

2.10 Explicit MPC

Due to the fact that online implicit MPC has a quite strict bound on computation time to work, the possibility of an offline explicit MPC is investigated. Using the "MPT" Toolbox again, a PWA system is defined according to Equation 17. The comfortability constraint is added and the cost function is defined as Equation 35. Then an implicit MPC is constructed using `MPCController()`, after which this is made explicit by `MPCController.toExplicit()`. This yields an explicit controller with as its inputs the current state and reference, and the output the optimal control.

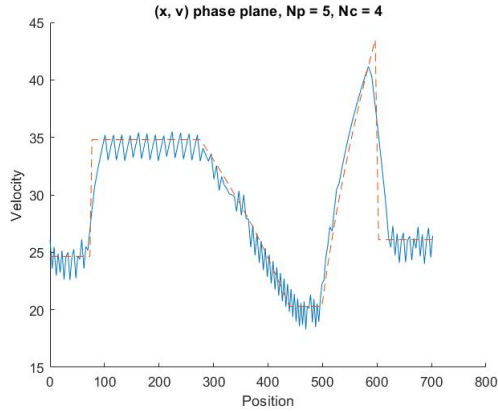
To compare the runtimes of both controllers, the exact same parameters are used. These are $N_c = N_p \in \{2, 3, 4\}$, $\lambda = 0.1$, a simulation time of 25 seconds, the time varying reference speed used before, and the runtime is only measured over the for-loop simulating real time. The average is taken of 30 runs, these are plotted in Figure 15.

It seems that the implicit method has a very slight increase in runtime based on the prediction horizon, whereas the explicit method seems to be increasingly slower. For longer prediction horizons these differences may become even larger.

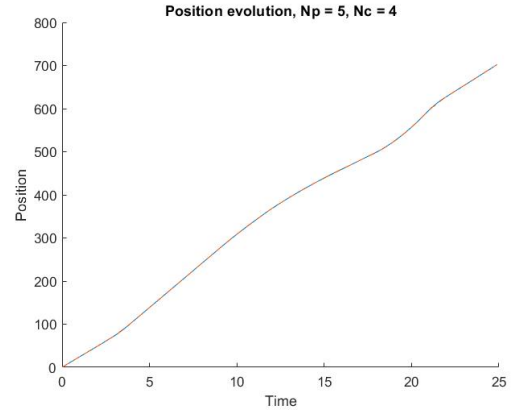
3 Conclusion

The aim of this section is to evaluate achieved results. In part 1, a simple hybrid system is considered. This model could be extended by including the physical mechanism of warming and cooling a device. If this dynamics are included and the temperature rate is also measured, an optimal MPC controller could be constructed for this problem in order to minimize energy consumption.

In the beginning of part 2, the friction dynamics are modeled in a simplified manner. Therefore, inevitably, an accumulative error is introduced. This is the case as the speed and friction force are correlated: an error in the friction force magnitude will result in an error in the speed magnitude and so on. It would therefore be interesting to consider the displacement progression of the car, such as visualized in figure 9b, as the displacement is the integral of the car speed and therefore also the integral of the speed error. At times 3 until 9 seconds the

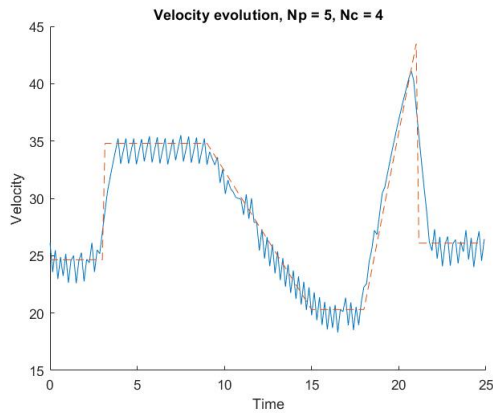


(a) Phase plot

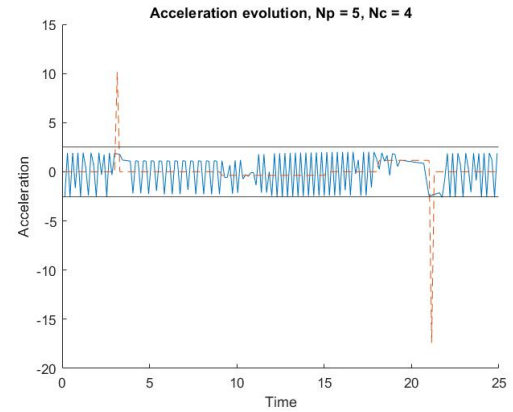


(b) Position evolution

Figure 9: Plots with the closed loop controller in blue, and the reference in striped orange



(a) Velocity evolution



(b) Acceleration with the comfortability boundary in black

Figure 10: Plots with the closed loop controller in blue, and the reference in striped orange

prescribed reference speed is constant at $v_{ref} = 1.2\alpha = 34.5 \frac{m}{s}$, as is visible in Figure 3 in the exercise description. This could mean a constant error and therefore two diverging curves in the position plot. Nevertheless, the model seems to slightly swing around the reference position. As can be seen in figure 16.

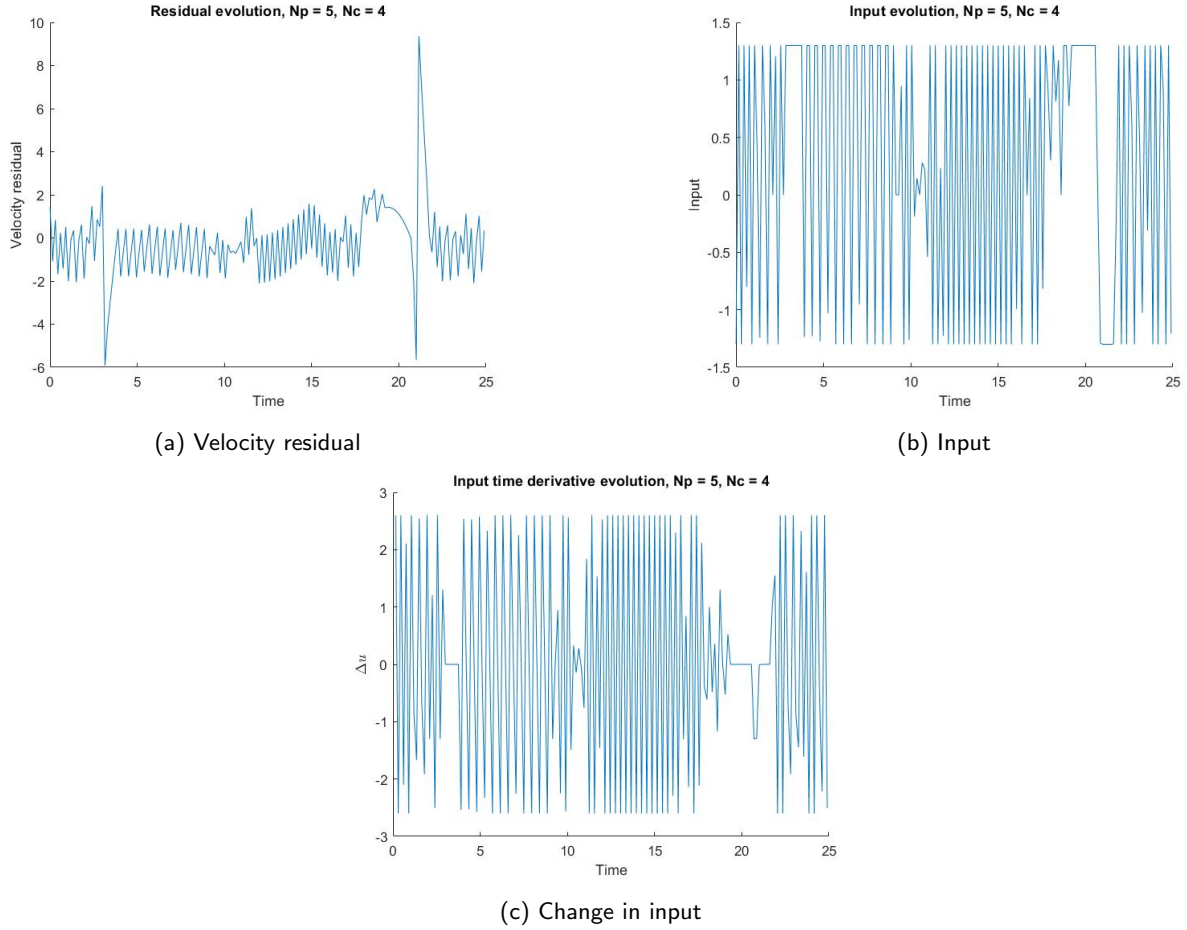


Figure 11: Plots with the closed loop controller

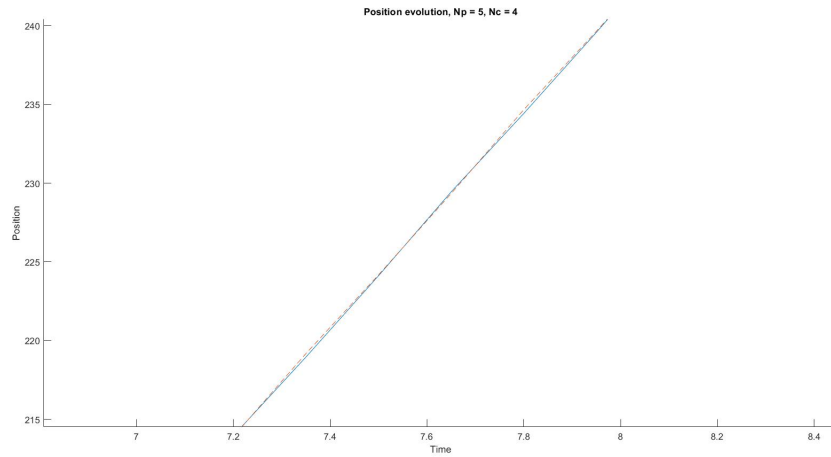
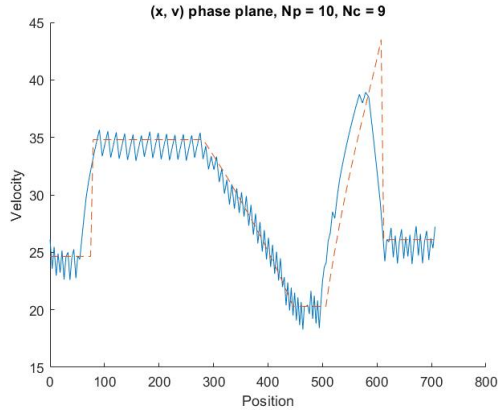
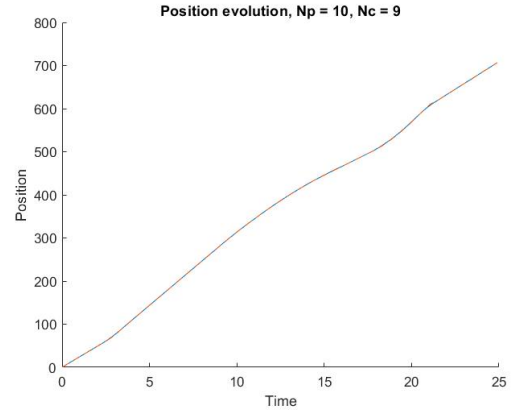


Figure 16: Close up of reference and modeled position values

The explanation for this phenomenon, has to do with the cross speed $v_{cross,2}$ which is very near the constant reference speed during this time interval. What happens, is that first the speed is modeled lower than in reality, as friction force is smaller than in reality according to figure 4, resulting in a positive speed error. The approximated

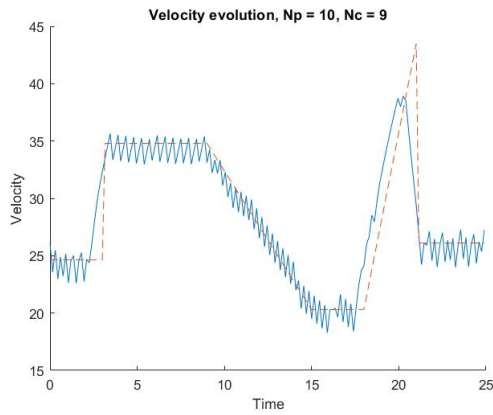


(a) Phase plot

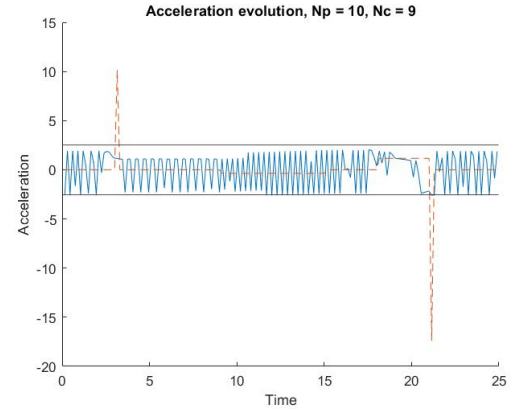


(b) Position evolution

Figure 12: Plots with the closed loop controller in blue, and the reference in striped orange



(a) Velocity evolution



(b) Acceleration with the comfortability boundary in black

Figure 13: Plots with the closed loop controller in blue, and the reference in striped orange

speed will increase more than in reality. However, as $v_{cross,2}$ will be reached rather quickly, after a short period of time, the approximated speed is larger than in reality, resulting in a negative speed error. This explains the oscillating behaviour of the model curve in figure 16.

Next the MLD model, which introduces some constraints of some constraints. The relevance of these can be investigated by looking at the constraint activity, possibly, some could be redundant and therefore left out for the sake of computational efficiency. Equation 21 gives the first implied constraints, which are obviously active, as the speed is at times lower and at times higher than α , thus a simplification can not be made here. Next, the speed boundaries described in equation 23 are in this specific case never active, but neither redundant, as for a different control input these boundaries will be achieved. The constraints describing the gear switches are very important and are often active in this case and finally the control boundaries, which related constraints are given in equation 28, are also not redundant, as there is no other constraint limiting the control input and in this case, there is being operated rather close to these limits. Furthermore, as mentioned, the resulting MLD model seems to be accurate, because figure 6 indicates that only around the gear switching velocities, there the MLD model differs from the PWA model. This difference is due to differences in the gear policy in the models at the switching points.

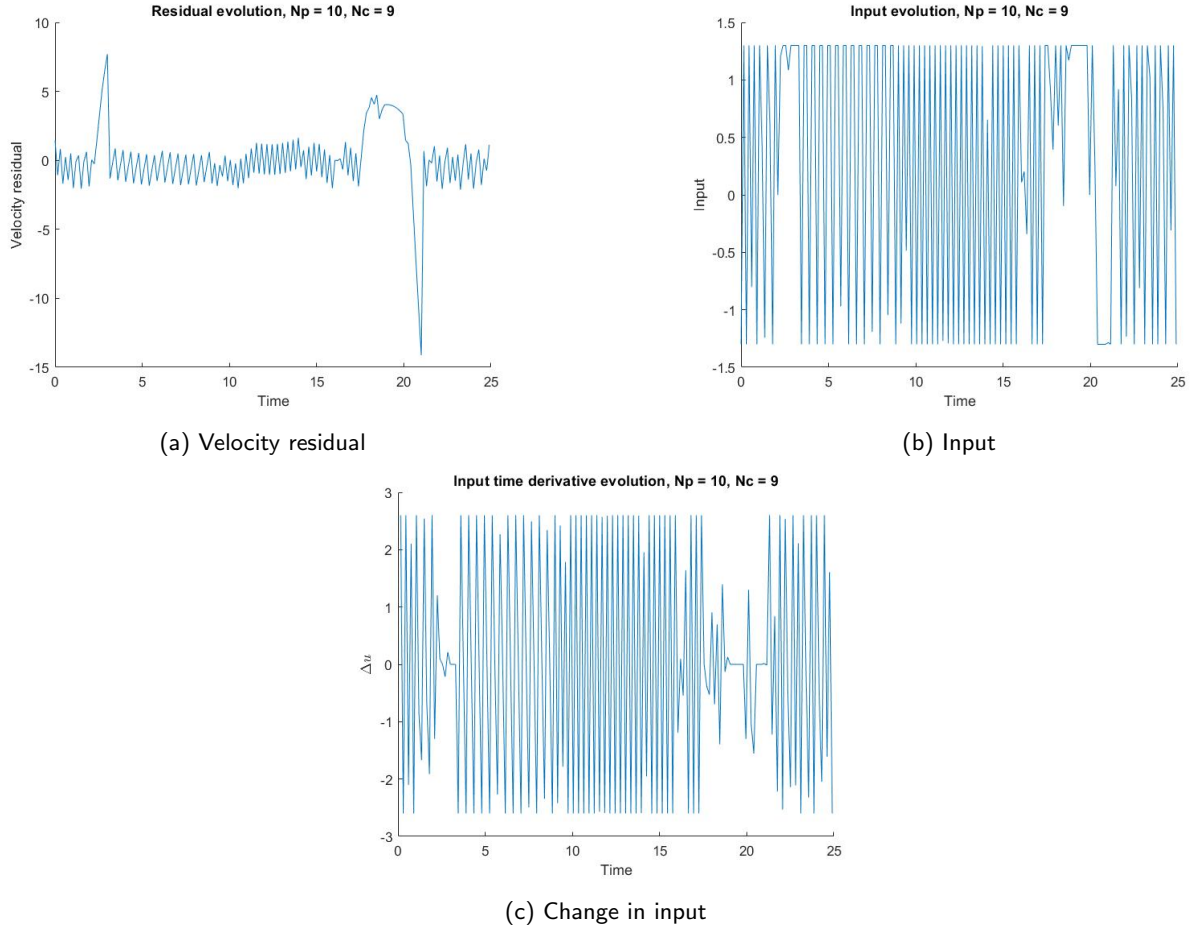


Figure 14: Plots with the closed loop controller

Now when the control loop is closed, the dynamics in figure 8 occur. The figure seems to be linearly decreasing until the equilibrium state in which the speed and input swing around v_{ref} and 0 respectively. By closer inspection, the velocity trend seems linearly decreasing, but this is not the case. The reason why this is less obvious than in the steady state part, is because the error in the predicted velocity is relatively slow with respect to the speed derivative. Therefore the shown velocity only swings slightly around this fictional line segment from the initial speed to the steady state value in the velocity plot.

Considering section 2.9, it strikes that the controller is able to make the model follow the reference speed accurately. A shorter time horizon does a better job yielding less over-/undershoot, comparing 11a and 14a. Besides, comparing figures 9a and 12a, it is clear that the longer time horizons introduce a larger lead with respect to the reference signals. The runtimes of the two models do not differ a lot, the model with the large time horizons is only 2-3% slower than the model that includes time horizons that are twice as short.

The investigation into the explicit MPC yielded little usable results. Only that the explicit method is significantly slower than the implicit method for small horizons. Thus it is concluded that in this case and for these horizons the implicit method is more time efficient than the explicit method.

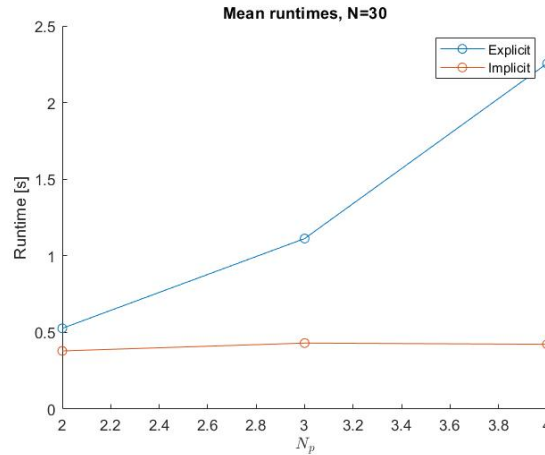


Figure 15: Runtime comparison between implicit and explicit MPC with different horizons

4 Appendix

Code

In this Appendix section, first the main code is presented. It makes use of function files, which are listed thereafter.

```

1 clc; close all; clear; tStart = tic;
2 variables % Retrieve system parameters
3 fig = 1; % Figure number token
4 WorkingOn = "all"; % Token to prevent always plotting everything
5 %% Dynamics test
6 u = @(t) u_max + heaviside(t-100)*(-u_max + u_min) + heaviside(t-120)*(-u_min + u_max)...
7     + heaviside(t-170)*(-u_max); % Input function
8 [t_0_1,y_0_1] = ode45(@(t,y) modelExact(t,y,u,vars, ""))...
9     , [0 t_end*step], [0 0]); % Integration of the exact model
10
11 % Plotting
12 if WorkingOn == "dyntest" || WorkingOn=="all"
13 figure(fig); fig = fig+1;
14 plot(t_0_1,y_0_1(:,2))
15 title 'ACC Car dynamics simulation'
16 xlabel 'time [s]'
17 ylabel 'speed [m/s]'
18 saveas(gcf,'Pics/Plot_2.1_1.jpg')
19
20 figure(fig); fig = fig+1;
21 plot(t_0_1,u(t_0_1))
22 title 'ACC Car dynamics simulation input'
23 xlabel 'time [s]'
24 ylabel 'speed [m/s]'
25 saveas(gcf,'Pics/Plot_2.1_2.jpg')
26 end
27
28 %% 2.1
29 v_max = ((b*u_max)/(c*(1+gamma*g(3))))^.5; % Maximum speed
30 a_max = (b*u_max)/(1+gamma*g(1))/m; % Maximum acceleration
31 a_min = (b*u_min)/(1+gamma*g(3))/m-c/m*v_max^2; % Minimum acceleration
32
33 vars.v_max = v_max; % Make struct to pass to functions later
34 %% 2.2
35
36 [alpha, beta] = optApprox(vars); % Determine optimal values for alpha and beta
37 vars.alpha = alpha; vars.beta = beta;
38

```

```

39 Vsamp = linspace(0, v_max, 200); % Speed samples
40 P_2 = fricApprox(Vsamp, vars); % Friction approximation
41
42 % Plotting
43 if WorkingOn == "2.2" || WorkingOn=="all"
44 figure(fig); fig = fig+1;
45 plot(Vsamp, P_2); hold on
46 plot(Vsamp, c*Vsamp.^2); hold off
47 title 'PWA approximation of friction force'
48 ylabel 'Friction force [N]'
49 xlabel 'speed [m/s]'
50 saveas(gcf, 'Pics/Plot_2.2.jpg')
51 end
52
53 %% 2.3
54 x0 = [0;0]; % Initial state
55 u = @(t) u_max/5 + u_max/2*sin(t/2/pi); % Input sinusoid
56 [t_3_1, y_3_1] = ode45(@(t,y) modelExact(t,y,u,vars, "gearlock")...
57     , [0 t_end*step], x0); % Integrate exact model
58 [t_3_2, y_3_2] = ode45(@(t,y) modelPWA(t,y,u,vars, "gearlock")...
59     , [0 t_end*step], x0); % Integrate PWA model
60
61 % Plotting
62 if WorkingOn == "2.3" || WorkingOn=="all"
63     fig = plot2_3(t_3_1, t_3_2, u, y_3_1, y_3_2, Vsamp, P_2, vars, fig);
64 end
65
66 %% 2.5
67 T = 1:t_end; % Define time vector
68 U = u(T*dt).*ones(size(T)); % Define input
69
70 [t_5_1, y_5_1] = FEuler(x0, u, vars); % Forward Euler integration of
71                                     % the PWA model
72 [t_5_2, y_5_2] = ode45(@(t,y) modelPWA(t,y,u,vars, "")...
73     , [0 t_end*dt], x0); % ode45 integration of PWA model
74
75 % Plotting
76 if WorkingOn == "2.5" || WorkingOn=="all"
77 figure(fig); fig = fig+1;
78 plot(t_5_2, u(t_5_2)); hold on
79 plot(T*dt, U, "--"); hold off
80 title 'Integration comparison input'
81 ylabel 'input'
82 xlabel 'time [s]'
83 legend("u(t)", 'Interpreter', 'latex')
84 saveas(gcf, 'Pics/Plot_2.5_1.jpg')
85
86 figure(fig); fig = fig+1;
87 plot(t_5_2, y_5_2(:, 2)); hold on
88 plot(t_5_1, y_5_1(2, :), "--"); hold off
89 title 'Integration comparison'
90 ylabel 'speed [m/s]'
91 xlabel 'time [s]'
92 legend("ODE45", "FE", 'Interpreter', 'latex')
93 saveas(gcf, 'Pics/Plot_2.5_2.jpg')
94 end
95
96 %% 2.6
97 sys = MLD(vars); % Create MLD model struct
98 % Unpack system
99 A = sys.A;
100 B1 = sys.B1; B2 = sys.B2; B3 = sys.B3;
101 E1 = sys.E1; E2 = sys.E2; E3 = sys.E3; E4 = sys.E4;
102 g5 = sys.g5;
103
104 H = [A B1 B2 B3];
105 nx = size(A, 1);
106 nu = size(B1, 2);

```

```

107 n = size(H, 2);
108
109 A = [[1 zeros(1, n-1)];... % v and u are as predefined
110      [0 1 zeros(1, n-2)];...
111      [E1 E2 E3 E4]]; % state is valid
112 C = [1 zeros(1, n-1)]; % Minimise the speed
113 ctype = ['SS', repmat('U', 1,size(E1,1))]; % Two equalities, rest upper bounds
114 vartype = 'CCBBBCCC'; % State either Continuous or Binary
115
116 V = 1:v_max; % Sample speeds
117 figure(fig); fig = fig+1;
118 subplot(2,2, 1:2); hold on
119 for u = u_min:0.2:u_max
120     u_ = @(t) u*t/t;
121     d = [];
122     for v = V
123         %MLD model
124         b = [v; u; g5];
125         [xMLD,~,status,~] = glpk (C,A,b,[],[],ctype,vartype);
126
127         xPmld = H*xMLD;
128
129         xPfe = [0;v] + vars.dt* modelPWA(1, [0;v], u_, vars, "");
130
131         d = [d xPmld - xPfe(2)];
132     end
133     plot(V, d)
134 end
135 xlabel("Speed")
136 ylabel("$v_{MLD} - v_{PWA}$", 'Interpreter', 'latex')
137
138 subplot(2,2, 3); hold on
139 V = 14.5:0.05:15.5; % Sample speeds
140 for u = u_min:0.2:u_max
141     u_ = @(t) u*t/t;
142     d = [];
143     for v = V
144         %MLD model
145         b = [v; u; g5];
146         [xMLD,~,status,~] = glpk (C,A,b,[],[],ctype,vartype);
147
148         xPmld = H*xMLD;
149
150         xPfe = [0;v] + vars.dt* modelPWA(1, [0;v], u_, vars, "");
151
152         d = [d xPmld - xPfe(2)];
153     end
154     plot(V, d, "x-")
155 end
156 xlabel("Speed")
157 ylabel("$v_{MLD} - v_{PWA}$", 'Interpreter', 'latex')
158
159 subplot(2,2, 4); hold on
160 V = 29.5:0.05:30.5; % Sample speeds
161 for u = u_min:0.2:u_max
162     u_ = @(t) u*t/t;
163     d = [];
164     for v = V
165         %MLD model
166         b = [v; u; g5];
167         [xMLD,~,status,~] = glpk (C,A,b,[],[],ctype,vartype);
168
169         xPmld = H*xMLD;
170
171         xPfe = [0;v] + vars.dt* modelPWA(1, [0;v], u_, vars, "");
172
173         d = [d xPmld - xPfe(2)];
174     end

```

```

175     plot(V, d, "x-")
176 end
177 xlabel("Speed")
178 ylabel("$v_{MLD} - v_{PWA}$", 'Interpreter', 'latex')
179
180
181
182 %% 2.7
183 % From here on the state x only consists of the speed v
184 x0 = 52; % Initial velocity
185 Np = 10; % Prediction horizon
186 Nc = 7; % Control horizon
187 lambda = 0.3; % Cost function lambda
188
189 sys = MLD(vars); % Create MLD model struct
190 sys.dt = dt;
191 sys.a_comf = 2.5; % Comfortability threshold
192 [F, b1, Neq, Nleq] = optContstraint(sys, Np, Nc); % All time invariant
193 % constraints regarding v, u, delta, z
194
195 vRef = 20 *ones(Np, 1); % Constant reference speed
196 [C, M, b2] = costFunc(sys, vRef, Np, lambda); % Get cost function and
197 % additional constraints for the auxilliary states
198 % originating from the norms
199
200 K = [[F, zeros(size(F,1),2*Np)];... % Compile constraints matrix
201      M];
202
203 L = [b1; b2]; % Compile constraints vector
204
205 if WorkingOn == "2.7" || WorkingOn=="all"
206     % Perform MPC and retrieve optimal input
207     [u_2_7, ~] = getOptInput(x0, vRef, sys, K, L, C, Np, Neq, Nleq, "plot");
208     fig = fig+1;
209 end
210
211 %% 2.8
212 T = 50; % Define number of timesteps
213 x0 = 55; % Initial state
214 vRef = 5 *ones(T+Np, 1); % Constant reference
215
216 Np = 10; % Prediction horizon
217 Nc = 7; % Control horizon
218 lambda = 0.3; % Cost function lambda
219
220 [F, b1, Neq, Nleq] = optContstraint(sys, Np, Nc); % Retrieve all time
221 % invariant constraints regarding v, u, delta, z
222
223 u_2_8 = zeros(T, nu);
224 x_2_8 = [x0; zeros(T-1, nx)];
225 for k = 1:T
226     vRef_k = vRef(k:k+Np-1); % Get reference at time k
227     [C, M, b2] = costFunc(sys, vRef_k, Np, lambda); % Get cost function
228
229     K = [[F, zeros(size(F,1),2*Np)];...
230          M]; % Compile constraints matrix
231
232     L = [b1; b2]; % Compile constraints vector
233
234     [u_2_8(k, :), ~] = getOptInput(x_2_8(k, :), vRef_k, sys, K, L, C...
235     , Np, Neq, Nleq, ""); % Get optimal input
236     x_2_8(k+1, :) = x_2_8(k, :) + modelExact(k*dt, [0;x_2_8(k, :)]...
237     , u_2_8(k, :), vars, "SingleState"); % Integrate timestep with input
238 end
239
240 % Plotting
241 if WorkingOn == "2.8" || WorkingOn=="all"
242     figure(fig); fig = fig+1;

```

```

243     subplot(2, 1, 1)
244     plot(x_2_8);
245     title("Velocities")
246
247     subplot(2, 1, 2)
248     plot(u_2_8);
249     title("Input")
250 end
251
252 %% 2.9
253 x0 = 0.9*alpha; % Define initial state
254
255 Tarr = 0:dt:25; % Create time array
256 N = max(size(Tarr)); % Number of timesteps
257
258 vRef = zeros(N,nx); % Create reference speed array
259 for k = 1:N
260     vRef(k) = vref(alpha, Tarr(k));
261 end
262 vRef = [vRef; vRef(end)*ones(Np,nx)];
263
264 lambda = 0.1; % Prediction horizon
265 Np = 5; % Prediction horizon
266 Nc = 4; % Control horizon
267
268 [F, b1, Neq, Nleq] = optContstraint(sys, Np, Nc); % All time invariant
269 %constraints regarding v, u, delta, z
270
271 u_2_9 = zeros(N, nu);
272 x_2_9 = [x0; zeros(N-1, nx)];
273 for k = 1:N % Same loop as in 2.8
274     vRef_k = vRef(k:k+Np-1);
275     [C, M, b2] = costFunc(sys, vRef_k, Np, lambda);
276
277     K = [[F, zeros(size(F,1),2*Np)];...
278         M];
279
280     L = [b1; b2];
281
282     [u_2_9(k, :), ~] = getOptInput(x_2_9(k, :), vRef_k, sys, K, L, C, Np, Neq, Nleq, "");
283     x_2_9(k+1, :) = x_2_9(k, :) + modelExact(k*dt, [0;x_2_9(k, :)], u_2_9(k, :)...
284         , vars, "SingleState");
285 end
286
287 if WorkingOn == "2.9" || WorkingOn=="all"
288     fig = plot2_9(0, x_2_9, u_2_9, vRef(1:N), sys.a_comf, Tarr, dt, fig, Np, Nc);
289 end
290
291 % Second Np, Nc combo, exactly the same as above otherwise
292 Np = 10; % Prediction horizon
293 Nc = 9; % Control horizon
294
295 vRef = zeros(N,nx);
296 for k = 1:N
297     vRef(k) = vref(alpha, Tarr(k));
298 end
299 vRef = [vRef; vRef(end)*ones(Np,nx)];
300
301 [F, b1, Neq, Nleq] = optContstraint(sys, Np, Nc); % All time invariant
302 %constraints regarding v, u, delta, z
303
304 u_2_9b = zeros(N, nu);
305 x_2_9b = [x0; zeros(N-1, nx)];
306 for k = 1:N
307     vRef_k = vRef(k:k+Np-1);
308     [C, M, b2] = costFunc(sys, vRef_k, Np, lambda);
309
310     K = [[F, zeros(size(F,1),2*Np)];...

```



```

311     M];
312
313     L = [b1; b2];
314
315     [u_2_9b(k, :), ~] = getOptInput(x_2_9b(k, :), vRef_k, sys, K, L, C, Np, Neq, Nleq, "");
316     x_2_9b(k+1, :) = x_2_9b(k, :) + modelExact(k*dt, [0;x_2_9b(k, :)], u_2_9b(k, :)...
317         , vars, "SingleState");
318 end
319
320 if WorkingOn == "2.9" || WorkingOn=="all"
321     fig = plot2_9(0, x_2_9b, u_2_9b, vRef(1:N), sys.a_comf, Tarr, dt, fig, Np, Nc);
322 end
323
324 %% 2.10
325
326 % Create explicit formulas
327 Np = 2;
328 explMPC2 = getexplMPCfun(vars, lambda, Np);
329
330 Np = 3;
331 explMPC3 = getexplMPCfun(vars, lambda, Np);
332
333 Np = 4;
334 explMPC4 = getexplMPCfun(vars, lambda, Np);
335
336 % Runtime counters
337 toc2 = 0; toc2i = 0; toc3 = 0; toc3i = 0; toc4 = 0; toc4i = 0;
338
339 for idx = 1:30
340 % Simulate with horizon 2
341     Np = 2;
342     vRef = zeros(N,nx);
343     for k = 1:N
344         vRef(k) = vref(alpha, Tarr(k));
345     end
346     vRef = [vRef; vRef(end)*ones(Np,nx)];
347
348     u_2_10a = zeros(N, nu);
349     x_2_10a = [x0; zeros(N-1, nx)];
350     tic
351     for k = 1:N
352         u_2_10a(k) = explMPC2.evaluate(x_2_10a(k, :), 'x.reference', vRef(k));
353         x_2_10a(k+1, :) = x_2_10a(k, :) + modelExact(k*dt, [0;x_2_10a(k, :)], u_2_10a(k, :)...
354             , vars, "SingleState");
355     end
356     toc2 = toc2 + toc;
357
358 % Implicit method
359 Np = 2; % Prediction horizon
360 Nc = 2; % Control horizon
361
362     vRef = zeros(N,nx);
363     for k = 1:N
364         vRef(k) = vref(alpha, Tarr(k));
365     end
366     vRef = [vRef; vRef(end)*ones(Np,nx)];
367
368     [F, b1, Neq, Nleq] = optContstraint(sys, Np, Nc); % All time invariant
369         %constraints regarding v, u, delta, z
370
371     u_2_10ai = zeros(N, nu);
372     x_2_10ai = [x0; zeros(N-1, nx)];
373     tic
374     for k = 1:N
375         vRef_k = vRef(k:k+Np-1);
376         [C, M, b2] = costFunc(sys, vRef_k, Np, lambda);
377
378     K = [[F, zeros(size(F,1),2*Np)];...

```

```

379     M];
380
381 L = [b1; b2];
382
383 [u_2_10ai(k, :), ~] = getOptInput(x_2_10ai(k, :), vRef_k, sys, K, L, C, Np, Neq, Nleq, "");
384 x_2_10ai(k+1, :) = x_2_10ai(k, :) + modelExact(k*dt, [0;x_2_10ai(k, :)], u_2_10ai(k, :)...
385     , vars, "SingleState");
386 end
387 toc2i = toc2i + toc;
388
389
390 % Simulate with horizon 3
391 Np = 3;
392 vRef = zeros(N,nx);
393 for k = 1:N
394     vRef(k) = vref(alpha, Tarr(k));
395 end
396 vRef = [vRef; vRef(end)*ones(Np,nx)];
397
398 u_2_10b = zeros(N, nu);
399 x_2_10b = [x0; zeros(N-1, nx)];
400 tic
401 for k = 1:N
402     u_2_10b(k) = explMPC3.evaluate(x_2_10b(k, :), 'x.reference', vRef(k));
403     x_2_10b(k+1, :) = x_2_10b(k, :) + modelExact(k*dt, [0;x_2_10b(k, :)], u_2_10b(k, :)...
404         , vars, "SingleState");
405 end
406 toc3 = toc3 + toc;
407
408
409 % Implicit method
410 Np = 3; % Prediction horizon
411 Nc = 3; % Control horizon
412
413 vRef = zeros(N,nx);
414 for k = 1:N
415     vRef(k) = vref(alpha, Tarr(k));
416 end
417 vRef = [vRef; vRef(end)*ones(Np,nx)];
418
419 [F, b1, Neq, Nleq] = optContstraint(sys, Np, Nc); % All time invariant
420     %constraints regarding v, u, delta, z
421
422 u_2_10bi = zeros(N, nu);
423 x_2_10bi = [x0; zeros(N-1, nx)];
424 tic
425 for k = 1:N
426     vRef_k = vRef(k:k+Np-1);
427 [C, M, b2] = costFunc(sys, vRef_k, Np, lambda);
428
429 K = [[F, zeros(size(F,1),2*Np)];...
430     M];
431
432 L = [b1; b2];
433
434 [u_2_10bi(k, :), ~] = getOptInput(x_2_10bi(k, :), vRef_k, sys, K, L, C, Np, Neq, Nleq, "");
435 x_2_10bi(k+1, :) = x_2_10bi(k, :) + modelExact(k*dt, [0;x_2_10bi(k, :)], u_2_10bi(k, :)...
436     , vars, "SingleState");
437 end
438 toc3i = toc3i + toc;
439
440
441 % Simulate with horizon 4
442 Np = 4;
443 vRef = zeros(N,nx);
444 for k = 1:N
445     vRef(k) = vref(alpha, Tarr(k));
446 end

```

```

447 vRef = [vRef; vRef(end)*ones(Np,nx)];
448
449 u_2_10c = zeros(N, nu);
450 x_2_10c = [x0; zeros(N-1, nx)];
451 tic
452 for k = 1:N
453     u_2_10c(k) = explMPC4.evaluate(x_2_10c(k, :), 'x.reference', vRef(k));
454     x_2_10c(k+1, :) = x_2_10c(k, :) + modelExact(k*dt, [0;x_2_10c(k, :)], u_2_10c(k, :)...
455         , vars, "SingleState");
456 end
457 toc4 = toc4 + toc;
458
459
460 % Implicit method
461 Np = 4; % Prediction horizon
462 Nc = 4; % Control horizon
463
464 vRef = zeros(N,nx);
465 for k = 1:N
466     vRef(k) = vref(alpha, Tarr(k));
467 end
468 vRef = [vRef; vRef(end)*ones(Np,nx)];
469
470 [F, b1, Neq, Nleq] = optContstraint(sys, Np, Nc); % All time invariant
471                                     %constraints regarding v, u, delta, z
472
473 u_2_10ci = zeros(N, nu);
474 x_2_10ci = [x0; zeros(N-1, nx)];
475 tic
476 for k = 1:N
477     vRef_k = vRef(k:k+Np-1);
478     [C, M, b2] = costFunc(sys, vRef_k, Np, lambda);
479
480     K = [[F, zeros(size(F,1),2*Np)];...
481         M];
482
483     L = [b1; b2];
484
485     [u_2_10ci(k, :), ~] = getOptInput(x_2_10ci(k, :), vRef_k, sys, K, L, C, Np, Neq, Nleq, "");
486     x_2_10ci(k+1, :) = x_2_10ci(k, :) + modelExact(k*dt, [0;x_2_10ci(k, :)], u_2_10ci(k, :)...
487         , vars, "SingleState");
488 end
489 toc4i = toc4i + toc;
490 end
491
492 % Average runtimes
493 toc2 = toc2/idx;
494 toc2i = toc2i/idx;
495 toc3 = toc3/idx;
496 toc3i = toc3i/idx;
497 toc4 = toc4/idx;
498 toc4i = toc4i/idx;
499
500 if WorkingOn == "2.10" || WorkingOn=="all"
501     figure(fig); fig = fig+1; hold on
502     plot([2 3 4], [toc2 toc3 toc4], "o-");
503     plot([2 3 4], [toc2i toc3i toc4i], "o-");
504     xlabel("$N_p$", 'Interpreter', 'latex')
505     ylabel("Runtime [s]")
506     title("Mean runtimes, N=30")
507     legend("Explicit", "Implicit")
508 end
509
510 tEnd = toc(tStart)

```

```

1 % Define all system paramaters as specified in the assignment
2 m = 800; % mass
3 c = 0.4; % friction constant

```

```

4 b = 3700; % driving constant
5 u_max = 1.3; % Maximum input
6 u_min = -1.3; % Minimum input
7 alphamax = 2.5; % Maximum acceleration
8 gamma = 0.87; % Gear constant
9 v12 = 15; % gear 2 switch
10 v23 = 30; % gear 3 switch
11
12 g = [1,2,3]; % gears
13 step=.1; % time step size
14 t_end = 3*10^3; % simulation time
15 dt = 0.15; % time step size for integration
16
17 % Make struct to pass to functions later
18 vars.m = m;
19 vars.c = c;
20 vars.b = b;
21 vars.u_max = u_max;
22 vars.u_min = u_min;
23 vars.gamma = gamma;
24 vars.v12 = v12;
25 vars.v23 = v23;
26 vars.g = g;
27 vars.t_end = t_end;
28 vars.dt = dt;
29 vars.a_comf = 2.5;

1 function [dx] = modelExact(t, y, u, vars, mode)
2 %MODELEXACT The exact differential equations
3
4 %% Determine u (either a function or a double)
5 if convertCharsToStrings(class(u)) == "function_handle"
6     ut = u(t);
7 else
8     ut = u;
9 end
10
11 %% Determine the gear
12 if mode == "gearlock"
13     gear = 1;
14 elseif y(2)<vars.v12
15     gear = 1;
16 elseif y(2)<vars.v23
17     gear=2;
18 else
19     gear=3;
20 end
21
22 %% Determine the derivative
23 if mode == "SingleState"
24     dx = 1/vars.m * vars.b/(1+vars.gamma*gear)*ut - 1/vars.m*vars.c*y(2)^2;
25 else
26     dx = [ y(2);...
27           1/vars.m * vars.b/(1+vars.gamma*gear)*ut - 1/vars.m*vars.c*y(2)^2];
28 end
29
30 end

1 function [alpha, beta] = optApprox(vars)
2 syms alph bet % Symbolic variable alpha and beta
3 syms v % Symbolic variable for the speed
4 c = vars.c; vmax = vars.v_max;
5
6 r1 = (c*v^2 - bet/alph *v)^2; % region 1
7 r2 = (c*v^2 - (c*vmax^2 - bet)/(vmax - alph) *v - bet + ...
8       (c*vmax^2 - bet)/(vmax - alph) *alph)^2; % region 2
9
10 A1 = int(r1, v, 0, alph); % Area 1

```

```

11 A2 = int(r2, v, alph, vmax); % Area 2
12
13 A = A1 + A2; % Total area
14
15 d_alph = diff(A, alph, 1); % deriv of A wrt to alpha
16 d_bet = diff(A, bet, 1); % deriv of A wrt to beta
17
18 dd_alph = diff(A, alph, 2); % double deriv of A wrt to alpha
19 dd_bet = diff(A, bet, 2); % double deriv of A wrt to beta
20
21 % Find extrema of A
22 [solAlph, solBet] = solve([d_alph == 0, d_bet == 0], [alph, bet]);
23 solAlph = double(solAlph);
24 solBet = double(solBet);
25
26 % Identify valid solutions
27 idx = find(solAlph>0 & solAlph<vmax & solBet>0 & solBet<(c*vmax^2));
28
29 alpha = solAlph(idx); % Grab valid values
30 beta = solBet(idx);
31
32 % Possibly plot the values of A dependent on alpha and beta
33 if 1==0
34     fA = matlabFunction(A); % Make a function of alpha, beta
35     figure()
36     fsurf(fA, [0 vmax 0 c*vmax^2])
37 end
38
39 % Confirm that the second derivative is positive
40 val_dd_alph = double(subs(dd_alph, [alph, bet], [alpha, beta]));
41 val_dd_bet = double(subs(dd_bet, [alph, bet], [alpha, beta]));
42
43 end

1 function [fig] = plot2_3(t_3_1,t_3_2, u, y_3_1,y_3_2, Vsamp, P_2, vars, fig)
2 %PLOT2_3 Make plots for Q2.3
3
4 %% Plot input
5 figure(fig); fig = fig+1;
6 plot(t_3_1,u(t_3_1));
7 title 'PWA approximation comparison input'
8 ylabel 'input'
9 xlabel 'time [s]'
10 % legend("Exact model", "PWA", 'Interpreter', 'latex')
11 saveas(gcf,'Pics/Plot_2.3_1.jpg')
12
13 %% Compare velocities
14 figure(fig); fig = fig+1;
15 plot(t_3_1,y_3_1(:, 2)); hold on
16 plot(t_3_2,y_3_2(:, 2)); hold off
17 title 'PWA approximation comparison'
18 ylabel 'speed [m/s]'
19 xlabel 'time [s]'
20 legend("Exact model", "PWA", 'Interpreter', 'latex')
21 saveas(gcf,'Pics/Plot_2.3_2.jpg')
22
23 %% Plot friction forces per speed, overlain with time evolution of models
24 figure(fig); fig = fig+1;
25 plot(Vsamp, P_2); hold on
26 plot(Vsamp, vars.c*Vsamp.^2);
27 plot(y_3_1(:, 2), t_3_1 .* P_2(end)/t_3_1(end));
28 plot(y_3_2(:, 2), t_3_2 .* P_2(end)/t_3_2(end)); hold off
29 title 'PWA approximation comparison'
30 ylabel 'Friction/time'
31 xlabel 'Speed'
32 legend("PWA friction", "Exact friction", "PWA evolution"...
33     , "Exact evolution", 'Interpreter', 'latex')
34 saveas(gcf,'Pics/Plot_2.3_3.jpg')

```

```

35
36 end

1 function [P] = fricApprox(v, vars)
2 %FRICAPPROX Approximates the friction force
3 % Symbol in report is P(v)
4
5 P = zeros(size(v));
6
7 boundaryIdx = find(v<vars.alpha, 1, 'last'); % Get switching point between functions
8 if all(v>vars.alpha) % or if every speed is above alpha
9     boundaryIdx = 0;
10 end
11
12
13 P(1:boundaryIdx) = vars.beta/vars.alpha .* v(1:boundaryIdx); % if v<alpha
14
15 P(boundaryIdx+1:end) = (vars.c*vars.v_max^2 - vars.beta)/(vars.v_max - vars.alpha)...
16     .* v(boundaryIdx+1:end) + vars.v_max/(vars.v_max-vars.alpha)*vars.beta ...
17     - vars.c*vars.v_max^2/(vars.v_max-vars.alpha)*vars.alpha;% if v>alpha
18
19 end

1 function [T, x] = FEuler(x0, u, vars)
2 %FEULER Forward Euler integration with initial condition x0 and input u
3
4 %% Initialisation
5 x = zeros(2, vars.t_end);
6 x(:, 1) = x0;
7 T = 1:vars.t_end;
8 T = T*vars.dt;
9
10 %% Integration
11 for t = 2:vars.t_end
12     x(:, t) = x(:, t-1) + vars.dt* modelPWA(t*vars.dt, x(:, t-1), u, vars, "");
13 end
14
15 end

1 function [sys] = MLD(vars)
2 %MLD Returns a struct with all of the MLD matrices
3
4 %% Unpack variables
5 dt = vars.dt;
6 m = vars.m;
7 a = vars.alpha;
8 uv = vars.v_max;
9 uu = vars.u_max;
10 lu = vars.u_min;
11 v12 = vars.v12;
12 v23 = vars.v23;
13 k = vars.beta/vars.alpha;
14 q = (vars.c*vars.v_max^2 - vars.beta)/(vars.v_max - vars.alpha);
15 r = vars.beta - q*vars.alpha;
16
17 %% Driving force
18 Fd = vars.b./(1+vars.gamma.*vars.g);
19
20 %% Define matrices
21 sys.A = 1 - dt*k/m;
22 sys.B1 = dt*Fd(1)/m;
23 sys.B2 = [0, 0, -dt*r/m];
24 sys.B3 = dt/m * [Fd(2)-Fd(1), Fd(3)-Fd(2), k-q];
25
26 sys.E1 = zeros(22, 1);
27 sys.E1([1 6 10 12 18]) = 1;
28 sys.E1([2 5 9 11 17]) = -1;
29

```

```

30 sys.E2 = zeros(22, 1);
31 sys.E2([3 16 22]) = 1;
32 sys.E2([4 15 21]) = -1;
33
34 sys.E3 = zeros(22, 3);
35 sys.E3(5:10, 3) = [a; -uv+a-eps; -uv; 0; 0; uv];
36 sys.E3(11:16, 1) = [v12; -uv+v12-eps; -uu; lu; -lu; uu];
37 sys.E3(17:22, 2) = [v23; -uv+v23-eps; -uu; lu; -lu; uu];
38
39 sys.E4 = zeros(22, 3);
40 sys.E4(7:10, 3) = [1; -1; 1; -1];
41 sys.E4(13:16, 1) = [1; -1; 1; -1];
42 sys.E4(19:22, 2) = [1; -1; 1; -1];
43
44 sys.g5 = zeros(22, 1);
45 sys.g5([1 10]) = uv;
46 sys.g5([3 16 22]) = uu;
47 sys.g5([4 15 21]) = -lu;
48 sys.g5(6) = a - eps;
49 sys.g5(12) = v12 - eps;
50 sys.g5(18) = v23 - eps;
51 end

1 function [dx] = modelPWA(t, y, u, vars, mode)
2 %MODELEXACT The piecewise affine approximation of the differential equations
3
4 %% Determine u (either a function or a double)
5 if convertCharsToStrings(class(u)) == "function_handle"
6     ut = u(t);
7 end
8
9 %% Determine the gear
10 if mode == "gearlock"
11     gear = 1;
12 elseif y(2) <= vars.v12
13     gear = 1;
14 elseif y(2) <= vars.v23
15     gear = 2;
16 else
17     gear = 3;
18 end
19
20 %% Determine the derivative
21 dx = [ y(2); ...
22     1/vars.m * vars.b/(1+vars.gamma*gear)*ut...
23     - 1/vars.m*fricApprox(y(2), vars)];
24
25 end

1 function [F, b, Neq, Nleq] = optContstraint(sys, Np, Nc)
2 %OPTCONSTRAINT Returns the time invariant constraint matrix F and vector b
3 %such that the constraints  $F x \sim b$  can be passed to the optimiser, where
4 %the relation  $\sim$  is to be specified in glpk.m. Neq is the number of ==
5 %constraints, and Nleq the number of <= constraints. F will begin with all
6 %of the == constraints, then the <=, then the >=.
7
8 %% Unpack system
9 A = sys.A;
10 B1 = sys.B1; B2 = sys.B2; B3 = sys.B3;
11 E1 = sys.E1; E2 = sys.E2; E3 = sys.E3; E4 = sys.E4;
12 g5 = sys.g5;
13
14 H = [A B1 B2 B3];
15 nx = size(A, 1);
16 nu = size(B1, 2);
17 n = size(H, 2);
18
19 aComf = sys.a_comf;

```

```

20 dt = sys.dt;
21
22 %% Create dynamic projection constraint (R x == 0)
23 T = kron(eye(Np), H); % Block diagonal with H
24
25 q = [eye(nx) zeros(nx, n-nx)]; % Extract x(k) for one timestep
26 Q = kron(diag(ones(Np-1,1),1),q); % Extract x(k+1) for all timesteps
27
28 R = T-Q; % x+ - H x = 0
29 R((end-nx+1):end,:) = []; % Remove zero row
30
31 %% Create logical constraints (E x <= G)
32 e = [E1 E2 E3 E4];
33 E = kron(eye(Np), e); % Block diagonal with E
34 G = kron(ones(Np, 1), g5); % Repeat g5 Np times
35
36 %% Create constant u constraint after Nc (W x == 0)
37 w = [zeros(nu, nx) eye(nu) zeros(nx, n-nx-nu)]; % Extract u(k) for one timestep
38 W = kron(eye(Np)-diag(ones(Np-1,1),-1),w); % u(k) - u(k+1) = 0
39 W = W(Nc:end, :); % Only from Nc and onwards
40
41 %% Guarantee comfortability constraint (P x <= p)
42 X = kron(eye(Np), q); % Extract all states
43
44 dX = Q - X; % x(k+1) - x(k)
45 dX(end, :) = []; % Last row is invalid
46 P = [-1/dt*dX;...
47      1/dt*dX]; % Double bound
48
49 p = aComf * ones(size(P,1), 1); % Constraint vector
50
51 %% Compile to F
52 F = [R; W; E; P];
53 b = [zeros(size([R;W],1),1); G; p];
54
55 %% Determine Neq, Nleq
56 Neq = size([R;W], 1); % Number of equality constraints
57 Nleq = size([E;P], 1); % Number of inequality constraints
58 end

1 function [c, M, b] = costFunc(sys, vRef, Np, lambda)
2 %COSTFUNC Return the c matrix for the minimisation problem min c' x with
3 %extra constraints M x <= b
4
5 %% Unpack system
6 A = sys.A;
7 B1 = sys.B1; B2 = sys.B2; B3 = sys.B3;
8 E1 = sys.E1; E2 = sys.E2; E3 = sys.E3; E4 = sys.E4;
9 g5 = sys.g5;
10
11 H = [A B1 B2 B3];
12 nx = size(A, 1);
13 nu = size(B1, 2);
14 n = size(H, 2);
15
16 %% The tracking cost
17 c_x = [zeros(1,Np*n) ones(1, Np) zeros(1,Np)]; % Sum rho_x
18
19 %% The input cost
20 c_u = lambda*[zeros(1,Np*n) zeros(1,Np) ones(1, Np)]; % Sum rho_u
21
22 %% The tracking constraints
23 x_ = [eye(nx) zeros(nx, n-nx)];
24 X = [kron(eye(Np), x_), zeros(Np*nx, 2*Np)]; % states
25
26 rhoX = [zeros(Np, Np*n) eye(Np) zeros(Np)]; % Extract rho_x from the state
27
28 M_x = [X - rhoX;...

```



```

29     -X - rhoX]; % Double bounded matrix
30
31     b_x = [vRef; -vRef]; % Double bounded vector
32
33     %% The input constraints
34     u_ = [zeros(nu, nx) eye(nu) zeros(nx, n-nx-nu)];
35     U = [kron(eye(Np), u_), zeros(Np*nx, 2*Np)]; % inputs
36
37     rhoU = [zeros(Np, Np*n) zeros(Np) eye(Np)]; % Extract rho_u from the state
38
39     M_u = [U - rhoU;...
40           -U - rhoU]; % Double bounded matrix
41
42     b_u = zeros(size(M_u, 1), 1); % Double bounded vector
43
44     %% Compile matrices (M x <= b)
45     c = c_x + c_u; % Cost function
46     M = [M_x; M_u]; % Auxilliary constraints matrix
47     b = [b_x; b_u]; % Auxilliary constraints vector
48
49     end

1 function [u, x] = getOptInput(x0, vRef, sys, K, L, C, Np, Neq, Nleq, plt)
2 %GETOPTINPUT Determine through MPC the optimal control input at time k
3 %% Unpack system
4 A = sys.A;
5 B1 = sys.B1; B2 = sys.B2; B3 = sys.B3;
6 E1 = sys.E1; E2 = sys.E2; E3 = sys.E3; E4 = sys.E4;
7 g5 = sys.g5;
8
9 H = [A B1 B2 B3];
10 nx = size(A, 1);
11 nu = size(B1, 2);
12 n = size(H, 2);
13
14 %% Add constraint x0 == x0 (0x == o)
15 O = [eye(nx), zeros(nx, size(K,2)-nx)]; % Matrix
16 o = x0; % Vector
17
18 %% Define and constrain terminal set ( |x_Np - xRef| <= TerSet )
19 TerSet = 58; % This is an inactive constraint by definition, since the
20              % whole of the state space is control invariant
21 S = [zeros(2*nx,n*(Np-1)) [eye(nx);-eye(nx)] zeros(2*nx,n-nx)...
22      zeros(2*nx,2*Np)]; % Grab the last state
23 s = TerSet * ones(2*nx, 1) + [eye(nx);-eye(nx)]*vRef(end); % Vector, double
24                          % bounded
25
26 %% Perform optimisation
27 ConstrA = [K;O;S]; % Compile constraint matrices
28 ConstrB = [L;o;s]; % Compile constraint vectors
29
30 ctype = [repmat('S', 1, Neq),... R and W ==
31          repmat('U', 1, Nleq+4*Np),... E, P and M <=
32          repmat('S', 1, nx),... Initial condition ==
33          repmat('U', 1, 2*nx)]; % Terminal set <=
34 vartype = [repmat('CCBBCCC', 1, Np), ... Np number of states
35            repmat('C', 1, nx*Np), ... nx*Np number of aux variables
36            repmat('C', 1, nu*Np)]; % nu*Np number of aux variables
37
38 % Perform optimisation
39 [xN,~,status,~] = glpk (C,ConstrA,ConstrB,[],[],ctype,vartype);
40 % status
41 %% Return first input and state
42 u = xN(nx+1:nx+nu);
43 x = xN(n+1:n+nx);
44
45 %% Possibly plot the predicted trajectory
46 if plt == "plot"

```

```

47     figure();
48     subplot(2, 1, 1)
49     x_ = [eye(nx) zeros(nx, n-nx)];
50     X = [kron(eye(Np), x_), zeros(Np*nx, 2*Np)]; % states
51     plot(X*xN);
52     title("Velocities")
53
54     subplot(2, 1, 2)
55     u_ = [zeros(nu, nx) eye(nu) zeros(nx, n-nx-nu)];
56     U = [kron(eye(Np), u_), zeros(Np*nx, 2*Np)]; % inputs
57     plot(U*xN);
58     title("Input")
59 end
60 end

1 function [fig] = plot2_9(x0, v, u, vRef, aComf, t, dt, fig, Np, Nc)
2 %PLOT2_9 Plot all of the plots asked for in Q2.9
3
4 N = max(size(t));
5 %% Integrate v to obtain x
6 % x+ = x + dt*v
7 dx = dt.*v;
8 dxRef = dt.*vRef;
9 x = [x0; zeros(N-1, 1)];
10 xRef = [x0; zeros(N-1, 1)];
11 for k = 1:N
12     x(k+1) = x(k) + dx(k);
13     xRef(k+1) = x(k) + dxRef(k);
14 end
15 x = x(1:end-1);
16 xRef = xRef(1:end-1);
17 v = v(1:end-1);
18
19 %% Plot phase plane
20 figure(fig); fig = fig+1; hold on
21 plot(x, v)
22 plot(xRef, vRef, "--")
23 xlabel("Position")
24 ylabel("Velocity")
25 title("(x, v) phase plane, Np = " + Np + ", Nc = " + Nc)
26 hold off
27
28 %% Plot x
29 figure(fig); fig = fig+1; hold on
30 plot(t, x)
31 plot(t, xRef, "--")
32 xlabel("Time")
33 ylabel("Position")
34 title("Position evolution, Np = " + Np + ", Nc = " + Nc)
35 hold off
36
37 %% Plot v
38 figure(fig); fig = fig+1; hold on
39 plot(t, v)
40 plot(t, vRef, "--")
41 xlabel("Time")
42 ylabel("Velocity")
43 title("Velocity evolution, Np = " + Np + ", Nc = " + Nc)
44 hold off
45
46 %% Plot acceleration
47 a = diff(v);
48 aRef = diff(vRef);
49 figure(fig); fig = fig+1; hold on
50 plot(t(2:end), a)
51 plot(t(2:end), aRef, "--")
52 yline(aComf, "k")
53 yline(-aComf, "k")

```

```

54 xlabel("Time")
55 ylabel("Acceleration")
56 title("Acceleration evolution, Np = " + Np + ", Nc = " + Nc)
57 hold off
58
59 %% Plot reference residual
60 figure(fig); fig = fig+1; hold on
61 plot(t, v-vRef)
62 xlabel("Time")
63 ylabel("Velocity residual")
64 title("Residual evolution, Np = " + Np + ", Nc = " + Nc)
65 hold off
66
67 %% Plot input
68 figure(fig); fig = fig+1; hold on
69 plot(t, u)
70 xlabel("Time")
71 ylabel("Input")
72 title("Input evolution, Np = " + Np + ", Nc = " + Nc)
73 hold off
74
75 %% Plot diff u
76 figure(fig); fig = fig+1; hold on
77 plot(t(2:end), diff(u))
78 xlabel("Time")
79 ylabel("$\Delta u$", Interpreter = 'latex')
80 title("Input time derivative evolution, Np = " + Np + ", Nc = " + Nc)
81 hold off
82 end

1 function [explMPC] = getexplMPCfun(vars, lambda, Np)
2 %% Unpack variables
3 alpha = vars.alpha;
4 beta = vars.beta;
5 b = vars.b;
6 c = vars.c;
7 gam = vars.gamma;
8 v_max = vars.v_max;
9 g = vars.g;
10 m = vars.m;
11 dt = vars.dt;
12 acomf = vars.a_comf;
13 v12 = vars.v12;
14 v23 = vars.v23;
15
16 %% Driving force
17 Fd = b./(1+gam.*g);
18
19 %% Friction approximation
20 k = beta/alpha;
21 q = (c*v_max^2 - beta)/(v_max - alpha);
22 r = beta - (c*v_max^2 - beta)/(v_max - alpha)*alpha;
23
24 %% Dynamics and systems
25 A1 = 1-dt*k/m; % v < alpha
26 A2 = 1-dt*q/m; % v >= alpha
27
28 f1 = 0;
29 f2 = -dt*r/m; % v >= alpha, constant
30
31 B = dt./m.*Fd; % Gears
32
33 C = 1;
34 D = 0;
35 g = 0;
36
37 sys1 = LTISystem('A', A1, 'B', B(1), 'C', C, 'D', D, 'f', f1, 'g', g, 'Ts', dt);
38 sys2 = LTISystem('A', A1, 'B', B(2), 'C', C, 'D', D, 'f', f1, 'g', g, 'Ts', dt);

```

```

39 sys5 = LTISystem('A', A2, 'B', B(2), 'C', C, 'D', D, 'f', f2, 'g', g, 'Ts', dt);
40 sys6 = LTISystem('A', A2, 'B', B(3), 'C', C, 'D', D, 'f', f2, 'g', g, 'Ts', dt);
41
42 %% Regions
43 X1 = Polyhedron([1;1;-1], [alpha-eps;v12-eps;0]);
44 X2 = Polyhedron([1;1;-1], [alpha-eps;v23-eps;-v12]);
45 X5 = Polyhedron([-1;1;-1], [-alpha;v23-eps;-v12]);
46 X6 = Polyhedron([-1;-1;1], [-alpha;-v23;v_max]);
47
48 % plot(X1, 'color', 'r', X2, 'color', 'b', X5, 'color', 'r', X6, 'color', 'b')
49
50 %% Link dynamics to regions
51 sys1.setDomain('x', X1);
52 sys2.setDomain('x', X2);
53 sys5.setDomain('x', X5);
54 sys6.setDomain('x', X6);
55
56 %% Construct system
57 sys = PWASystem([sys1, sys2, sys5, sys6]);
58
59 %% Add constraints
60 sys.x.min = 0;
61 sys.x.max = v_max;
62 sys.u.min = vars.u_min;
63 sys.u.max = vars.u_max;
64 sys.x.with('deltaMin');
65 sys.x.with('deltaMax');
66 sys.x.deltaMin = -acomf; % Comfortability constraint
67 sys.x.deltaMax = acomf;
68
69 sys.x.with('reference'); % The state is a tracking problem
70 sys.x.reference = 'free';
71
72 %% Determine cost
73 sys.u.penalty = OneNormFunction( lambda );
74 sys.x.penalty = OneNormFunction( 1 );
75
76 %% Make Explicit controller
77 implMPC = MPCController(sys, Np);
78
79 explMPC = implMPC.toExplicit(); % Make explicit
80 end

```