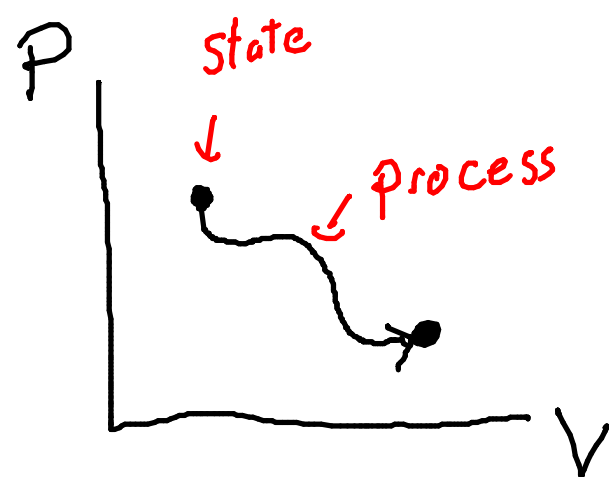


Chapter 4

Heat Engines &

Refrigerators



ideal gas

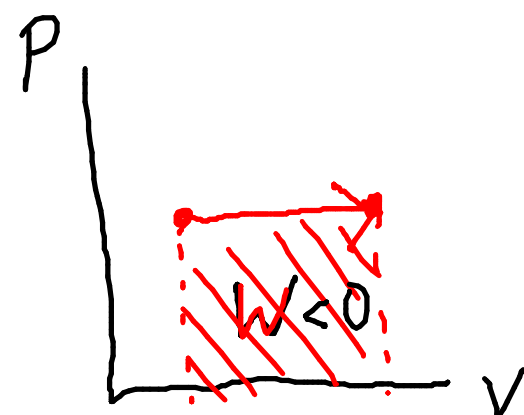
$N$  constant

$$PV = NkT$$

$T$  is determined by  
 $P$  &  $V$ .

$$U = N \frac{f}{2} kT$$

same pressure  
isobaric expansion



atmospheric pressure



ideal piston

free to move

ideal gas  
is in equilibrium  
with atmosphere

$$PV = NkT$$

const  $\uparrow$   $\uparrow$

expansion: work is flowing out

$$W = -P\Delta V = -\int P dV$$

$$W < 0$$

$W <$  area under a process

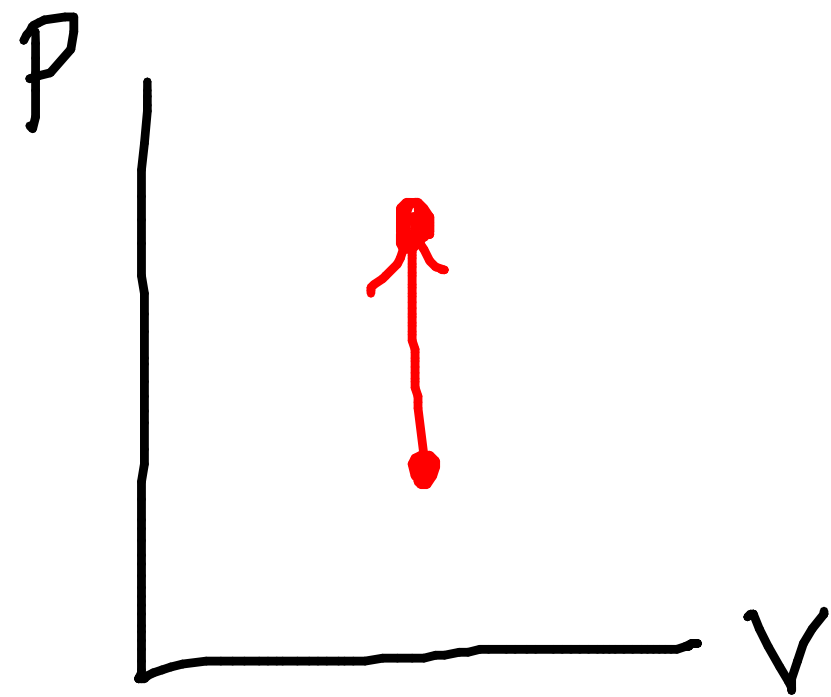
process  $\rightarrow$

process  $\leftarrow$

$$W < 0$$

$$W > 0$$

isometric / isochoric process  
"constant volume" process



$$PV = NkT$$

↑ const      ↑

$$W = 0$$

$$\Delta U = N \frac{f}{2} k \Delta T > 0$$

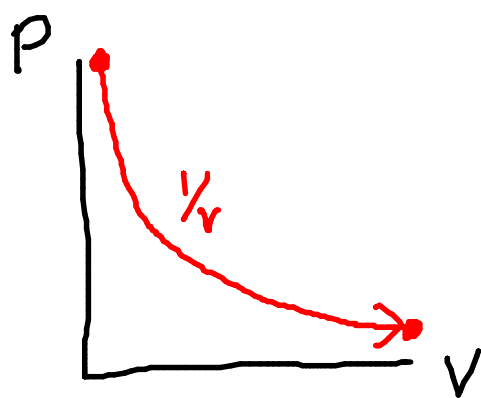
$$Q > 0 \quad \text{heat flows in}$$

isothermal process

$$T = \text{constant}$$

$$PV = NkT = \text{constant}$$

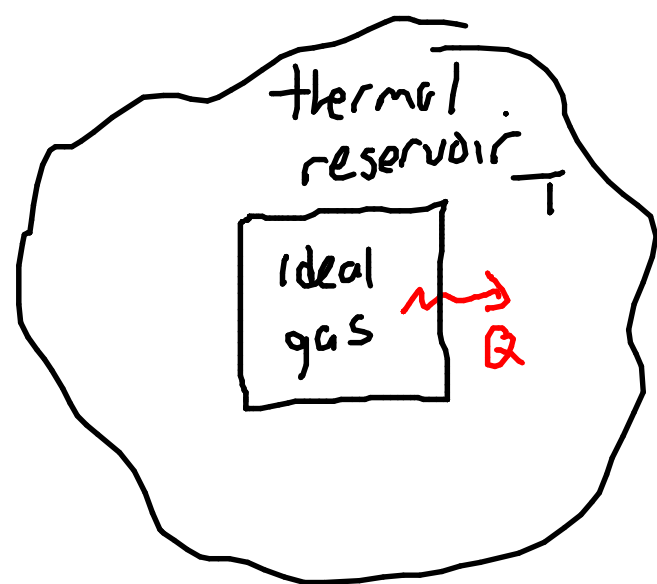
$$P = \frac{C}{V}$$



$$\Delta U = 0$$

$$W > 0$$

because it's expanding  
 $\therefore Q < 0$  heat flows out



must stay in thermal  
equilibrium with reservoir  
 $\therefore$  isothermal process is  
relatively slow

$$W = - \int P dV = - \int \frac{NkT}{V} dV = -NkT \int \frac{dV}{V}$$

$$= -NkT \ln V \rightarrow -NkT \ln \frac{V_f}{V_i}$$

$$\Delta U = 0$$

$$Q = -W = +NkT \ln \frac{V_f}{V_i}$$

Adiabatic Process: no heat flow, or no change in entropy

- rather quickly so no heat flows  
(insulation helps)
- not too fast or else you'll get turbulence &  $S$  increases  
quasi-static but fast

$$Q = 0 \rightarrow dU = W = -P dV$$

$$\frac{f}{2} N k dT = -P dV$$

$$\frac{f}{2} N k dT = -\frac{N k T}{V} dV$$

$$\int_{T_i}^{T_f} \frac{f}{2} \frac{dT}{T} = - \int_{V_i}^{V_f} \frac{dV}{V}$$

$$\frac{f}{2} \ln \frac{T_f}{T_i} = - \ln \frac{V_f}{V_i}$$

$$\left( \frac{T_f}{T_i} \right)^{f/2} = \left( \frac{V_f}{V_i} \right)^{-1}$$

$$\rightarrow V_f T_f^{f/2} = V_i T_i^{f/2}$$

$$\rightarrow V T^{f/2} = \text{constant}$$

$$\rightarrow [V(PV)^{f/2}]^{2/f} = [\text{constant}]^{2/f}$$

$$\rightarrow P V^{1 + \frac{2}{f}} = \text{constant}$$

$$\boxed{P V^{\frac{f+2}{f}} = \text{constant}}$$

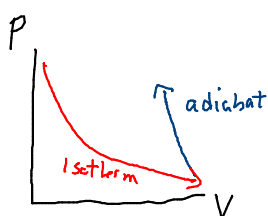
$$\gamma = \frac{f+2}{f} \quad \text{adiabatic exponent}$$

$$f=1 \quad \gamma=3$$

$$f=3 \quad \gamma=\frac{5}{3}$$

...

$$f \rightarrow \infty \quad \gamma=1$$



e.g. Diesel engine quickly compresses air

to  $\frac{1}{20}$ th of initial value.

What is  $T$  after compression?

$$T_f^{f/2} V_f = T_i^{f/2} V_i$$

$$T_f V_f^{2/f} = T_i V_i^{2/f}$$

$$T_f = T_i \left( \frac{V_i}{V_f} \right)^{2/f}$$

$$T_f = (300K) (20)^{2/5}$$

$$\approx 1000K \text{ or } 700^\circ C$$

$$T_i = 300K$$

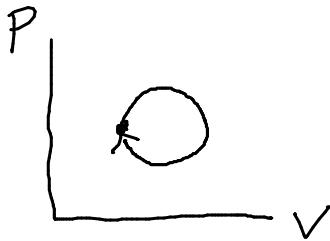
$$V_f = V_i \frac{1}{20}$$

$$\frac{V_i}{V_f} = 20$$

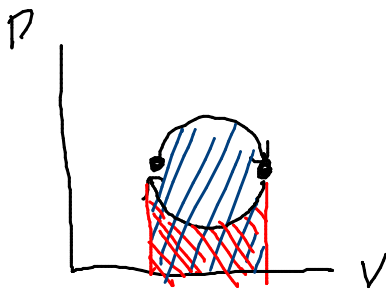
$$f=5$$

Diesel engines can ignite gas without sparks

# Cyclic Process



After each cycle, all state variables must return to original variables especially the entropy



work done on bottom half

$$W_b > 0$$

work done on top half

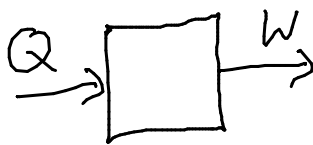
$$W_t < 0$$

$$|W_t| > |W_b|$$

$$W = W_t + W_b < 0$$

$U$  is a state variable,  
so if net work flows out of system  
after one cycle, heat must flow in

perpetual motion  
1st kind



heat is converted to work

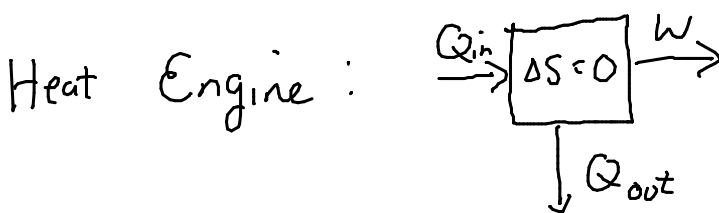
perpetual motion machine  
of the 2nd kind - impossible

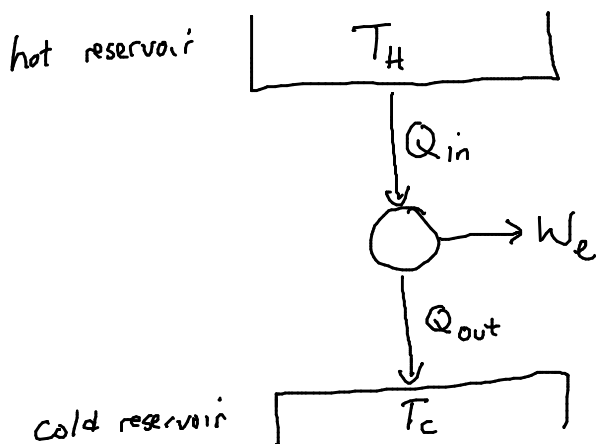
why?

$Q$  flows in, entropy increases

in a cyclic process,

entropy has to return to its initial value,  
so excess entropy must be disposed of





$$Q_{in} = W_e + Q_{out}$$

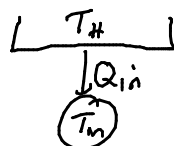
Efficiency of Engine

$$\eta = \frac{W_e}{Q_{in}} \leq 100\%$$

$$= \frac{Q_{in} - Q_{out}}{Q_{in}} = 1 - \frac{Q_{out}}{Q_{in}}$$

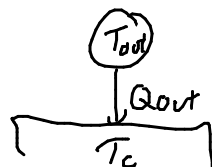
Let  $T_{in}$  be temperature of engine when heat flows in  
 $T_{in} < T_H$

$$\Delta S = + \frac{Q_{in}}{T_{in}}$$



Let  $T_{out}$  be temperature of engine when heat flows out  
 $T_{out} > T_C$

$$\Delta S = - \frac{Q_{out}}{T_{out}}$$



$$\Delta S = \frac{Q_{in}}{T_{in}} - \frac{Q_{out}}{T_{out}} = 0$$

$$\frac{Q_{in}}{Q_{out}} = \frac{T_{in}}{T_{out}} \rightarrow \eta = 1 - \frac{T_{out}}{T_{in}}$$

To maximize  $\eta$ ,

- make  $T_{out}$  small, but  $T_{out} > T_C$
- make  $T_{in}$  big, but  $T_{in} < T_H$

Best we  
can  
do

$$\eta < \underbrace{1 - \frac{T_C}{T_H}}_{\text{maximum possible efficiency}} = \frac{T_H - T_C}{T_H}$$

make this small

e.g.  $T_C = 300K$   $T_H = 1000K \approx 700^\circ C$

$$\eta < 70\%$$

$$\eta = 1 - \frac{T_{out}}{T_{in}}$$

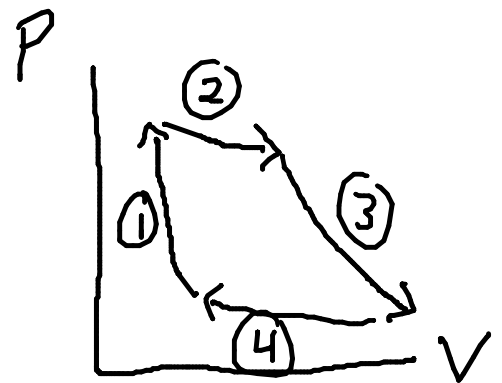
Carnot engine is maximally efficient

bring  $T_{out} \gtrsim T_c$

$T_{in} \lesssim T_H$

- 1) raise  $T$  of engine to almost  $T_H$ , adiabatically (no heat, fast)
- 2) let heat flow from hot reservoir into engine almost isothermal
- 3) lower  $T$  to almost  $T_c$ , adiabatically
- 4) let heat flow out of engine (isothermal)

$$\eta_{carnot} = 1 - \frac{T_c}{T_H}$$



Steps 2 & 4 are isothermal - slow

When heat flows, rate of flow depends on  $\Delta T$

$\Delta T$  is <sup>very</sup> small, rate of flow is very slow

→ low-power, high-efficiency engine