

## **GATE ESE 2020 TARGET ECE ENGINEERING**

## **GATE ESE PSU's 2019-20**

## **GATEACADEMY**

### **103 CONTROL SYSTEM\_GATEACADEMY**

**TOTAL PAGE CONTROL SYSTEM-710 PGAE**

### **LATEST GATEACADEMY CLASSROOM HANDWRITTING**

#### **CONTENT COVERED:**

- 1. Theory Notes**
- 2. Explanation**
- 3. Derivation**
- 4. Example**
- 5. Shortcut & Formula Summary**
- 6. Previous year Paper Q. Sol.**

**Noted:- Single Source Follow, Revise**

**Multiple Time Best key of Success**

- Syllabus
1. Basic Concept of Control System
  2. Block Diagram and Signal flow graph
  3. Time Response
  4. Routh Hurwitz stability
  5. Root locus
  6. Polar and Nyquist plot
  7. Bode plot
  8. frequency Response of Second order System
  9. Controller & Compensator
  10. state space Analysis

Reference Book

1. Automatic Control System by kuo
2. Modern Control Engineering by Ogata

## Chapter 01 Basic Concept of Control System

System :- System is a group of Component which gives proper output for a given Input.

Control System :- Control System is a group of Component which give controlled output for a given Input.

Classification of Control Systems depends on Controlling Mechanism in the System. There are two types of Control Systems.

1. Open Loop Control System

2. Closed Loop Control System



fig. open loop control system / open system

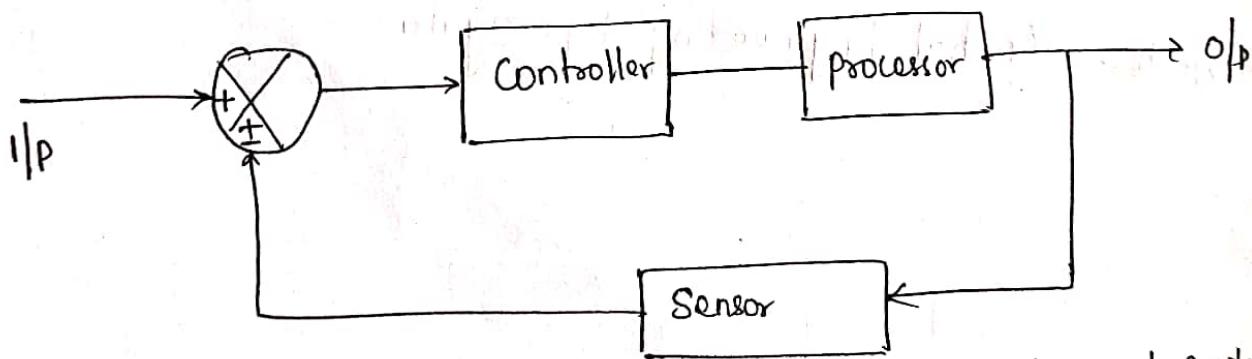


fig. closed loop control system / closed system

Key points:

1. In case of open loop control system or open system, controlling action does not depends on output.
2. In case of closed loop control system controlling action depends on output.

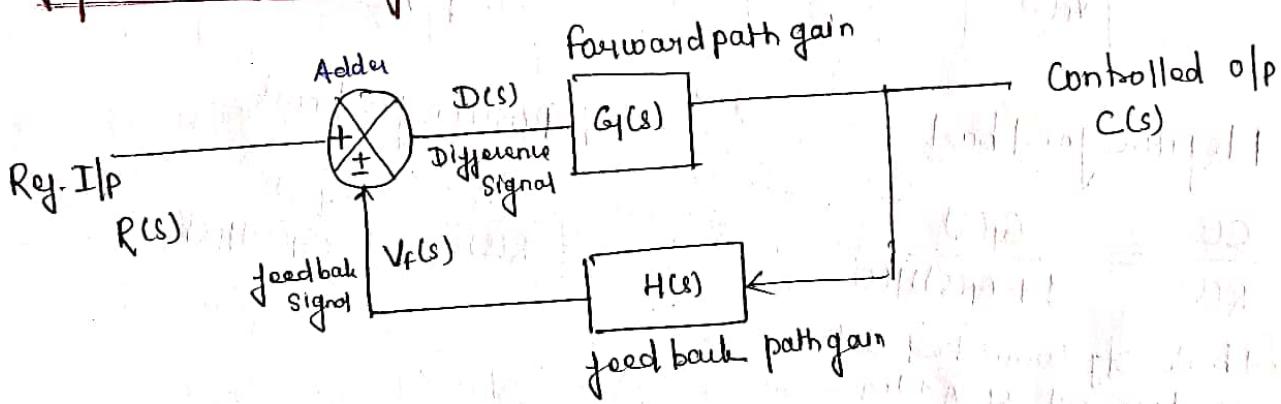
## Examples

1. fan without regulator is a System.
2. fan with regulator is an open loop control system.
3. Air Conditioner (AC) is a closed loop control system.
4. Normal iron box is an open loop control system.
5. Automatic iron box is a closed loop control system.
6. Electric lamp is an open loop control system.
7. Traffic light Signal is an open loop control system.

## Key point

1. All devices having Sensor are closed loop control systems.

## \* Representation of Closed Loop Control System:



$$G_f(s) = \frac{C(s)}{D(s)} \quad \text{--- (1)}$$

$$H(s) = \frac{V_f(s)}{C(s)} \quad \text{--- (2)}$$

$$D(s) = R(s) \pm V_f(s) \quad \text{--- (3)}$$

$$\frac{C(s)}{R(s)} = T(s) = \frac{C(s)}{R(s) + V_f(s)} \quad \text{closed loop transfer function (CLTF)}$$

from eq(1)

$$C(s) = G_f(s) D(s)$$

$$\text{from eq(3)} \quad C(s) = G_f(s) [R(s) \pm V_f(s)]$$

$$\text{from eq(2)} \quad C(s) = G_f(s) [R(s) \pm H(s) C(s)]$$

$$\rightarrow C(s) [1 + G(s) H(s)] = R(s) G(s)$$

$$\frac{C(s)}{R(s)} = \frac{G(s)}{1 + G(s) H(s)}$$

Transfer function



Negative feed back

$$\frac{C(s)}{R(s)} = \frac{G(s)}{1 + G(s)H(s)}$$

feed back sig connected with -ve terminal of Adder

Positive feed back

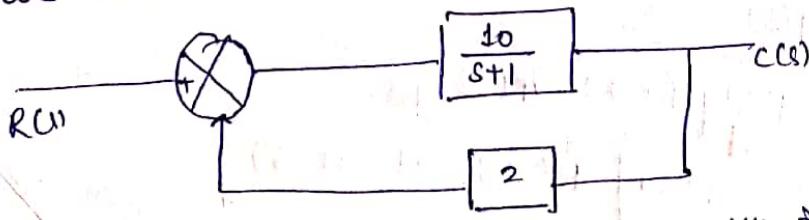
$$\frac{C(s)}{R(s)} = \frac{G(s)}{1 - G(s)H(s)}$$

key points

oscillator and Multivibrator

1. Positive feedback is used for designing of Amplifier.
2. Negative feedback is used for designing of systems.
3. positive feedback decreases the stability of systems.
4. Negative feedback increases the stability of systems.
5. Positive feedback system is referred as regenerative feed back system
6. Negative feedback system is referred as degenerative feed back system
7. If information of feedback (+ve/-ve) is not given then we will consider negative feedback system.

e.g.



$$\frac{C(s)}{R(s)} = \frac{10}{s+1} \cdot 2 = \frac{10}{s+2}$$

8. In case of unity feedback system we consider  $H(s) = 1$

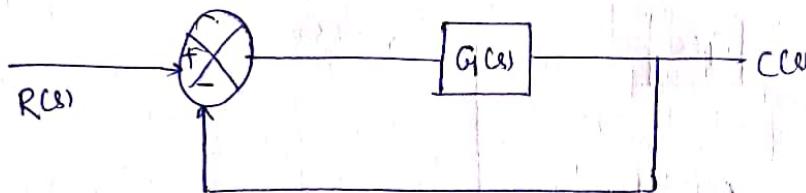


fig. Negative unity feedback system

$$T(s) = \frac{C(s)}{R(s)} = \frac{G_1(s)}{1 + G_1(s)}$$

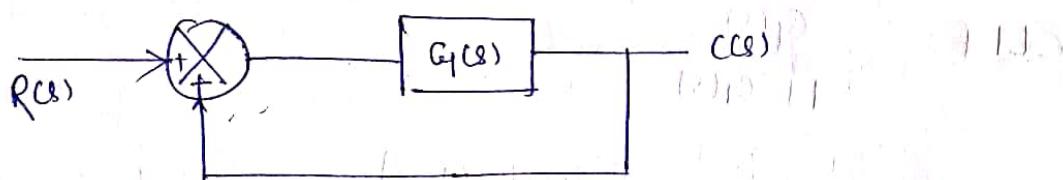


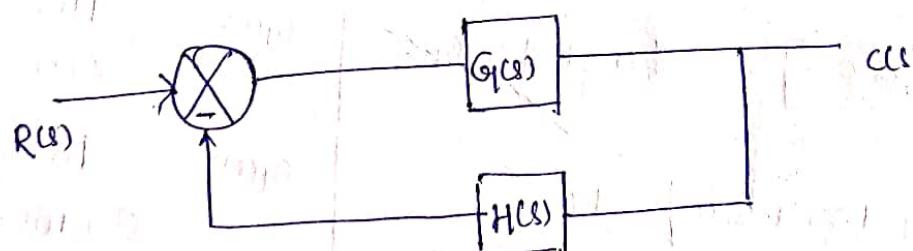
fig. positive unity feedback system

$$T(s) = \frac{C(s)}{R(s)} = \frac{G_1(s)}{1 - G_1(s)}$$

9. If information of feedback element  $H(s)$  is not given then we will consider unity feedback system that means  $H(s) = 1$ .

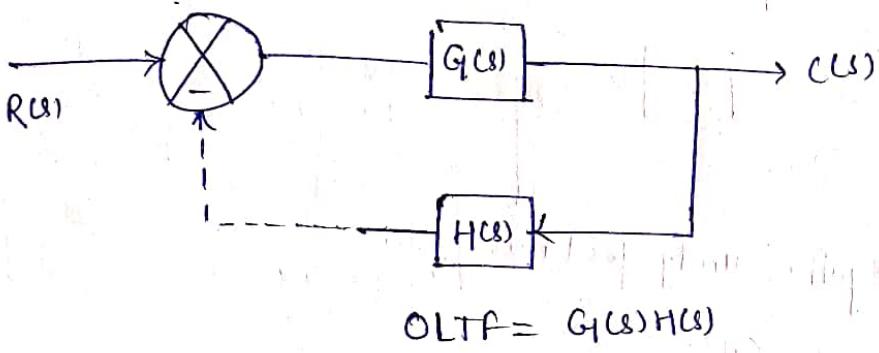
10. Mostly we use negative unity feedback system for analysis of control systems.

#### \* Open loop Transfer function :



$$OLTF = \frac{G_1(s)}{1 + G_1(s)H(s)}$$

$G_1(s)H(s) =$  open loop transfer function



In Case of Negative Unity feed back System

$$CLTF = \frac{G(s)}{1 + G(s)}$$

$$OLTF = G(s) \quad [H(s) = 1]$$

$$CLTF = \frac{OLTF}{1 + OLTF}$$

only in case of unity negative feedback systems

$$\text{eg. } T(s) = \frac{10}{s^2 + 10s + 30}$$

negative unity feed back

$$\frac{G(s)}{1 + G(s)} = \frac{10}{s^2 + 10s + 30}$$

$$\Rightarrow G(s)[s^2 + 10s + 30] = 10 + 10G(s)$$

$$\Rightarrow G(s)[s^2 + 10s + 20] = 10$$

$$G(s) = \frac{10}{s^2 + 10s + 20}$$

$$\frac{G(s)}{1 + G(s)} = \frac{10}{s^2 + 10s + 30}$$

$$G(s) = \frac{H(s)}{G(s) - H(s)}$$

$$G(s) = \frac{10}{s^2 + 10s + 20}$$

DC gain of CLTF :

DC = zero freq.

$$f=0$$

$$\omega=0$$

$$s=0$$

$$T(s=0) = \frac{10}{0+0+30} = \frac{1}{3}$$

## DC gain of open loop Transfer function

$$G(s) = \frac{10}{s^2 + 10s + 20}$$

$$G(s=0) = 0.5$$

Q.  $G(s) = \frac{10}{s(s+2)}$  DC gain =  $\infty$  X (wrong)

Sensitivity :-

$$A = f(B)$$

$$\boxed{S_B^A = \frac{\partial A/A}{\partial B/B} = \frac{B}{A} \times \frac{\partial A}{\partial B}}$$

$$T = \frac{G}{1+GH}$$

$$T = f(G, H)$$

i)  $S_G^T = \frac{\partial T/T}{\partial G/G} = \frac{G}{T} \times \frac{\partial T}{\partial G}$  ①

ii)  $S_H^T = \frac{\partial T/T}{\partial H/H} = \frac{H}{T} \times \frac{\partial T}{\partial H}$  ②

$$\frac{\partial T}{\partial G} = \frac{(1+GH) \times 1 - G(0+H)}{(1+GH)^2}$$

$$= \frac{1+GH-GH}{(1+GH)^2}$$

$$\frac{\partial T}{\partial G} = \frac{1}{(1+GH)^2}$$

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**Multiple Time Best key of Success**

$$S_{G_1}^T = \frac{\frac{G_1}{G}}{1+G_1H} \times \frac{1}{(1+G_1H)^2}$$

$$\boxed{S_{G_1}^T = \frac{1}{1+G_1H}}$$

now.  $S_H^T = \frac{H}{\frac{G}{1+G_1H}} \times \frac{\partial T}{\partial H}$

$$\frac{\partial T}{\partial H} = \frac{(1+G_1H)^2 - G_1^2}{(1+G_1H)^2}$$

$$= \frac{-G_1^2}{(1+G_1H)^2}$$

$$S_H^T = \frac{H}{\frac{G}{1+G_1H}} \times \left( \frac{-G_1^2}{(1+G_1H)^2} \right)$$

$$\Rightarrow \frac{-HG_1}{(1+G_1H)}$$

$$\boxed{S_H^T = \frac{-GH}{1+G_1H}}$$



$$T = \frac{C}{R} = G$$

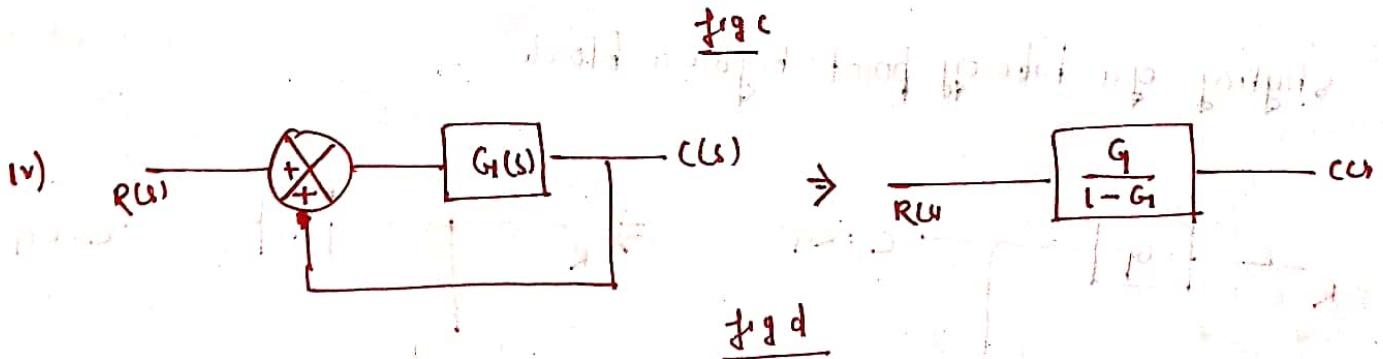
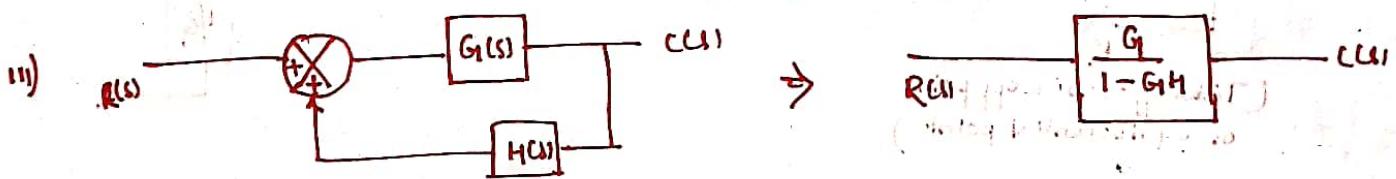
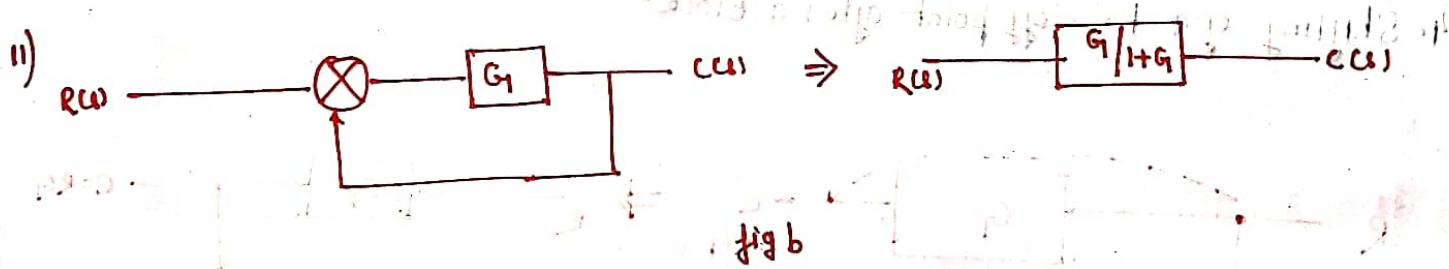
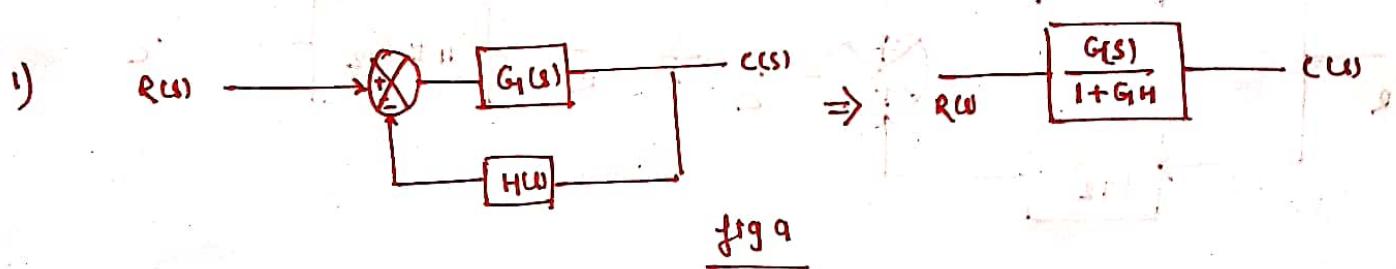
$$S_G^T = \frac{\partial T|_T}{\partial G_1|_G_1} = \frac{G_1}{T} \times \frac{\partial T}{\partial G_1}$$

## Block Diagram

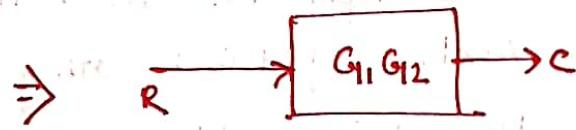
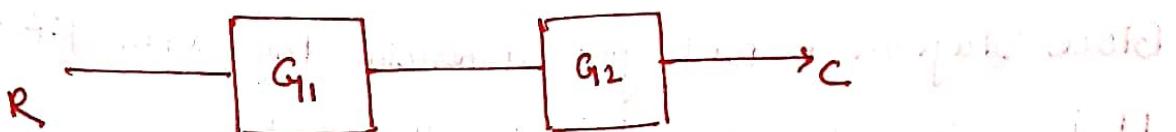
1. Block Diagram is used for determine the overall T.f of the system.
2. It is a pictorial representation of the system.
3. Analysis of T.f by using Block diag. is based on Block Diag. Reduction rule.
4. In a Block diag. representn all system Variable are linked to each other through functional block.

### \* Block Diagram Reduction Rules:

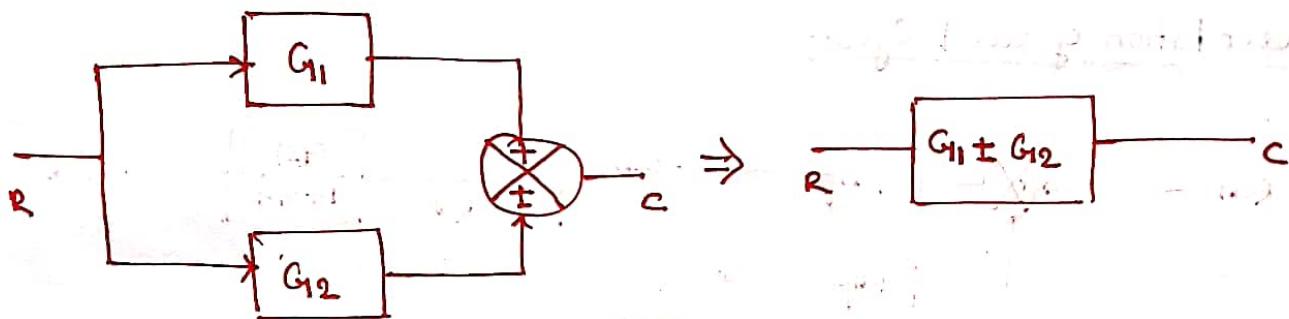
#### i. Representation of closed System:



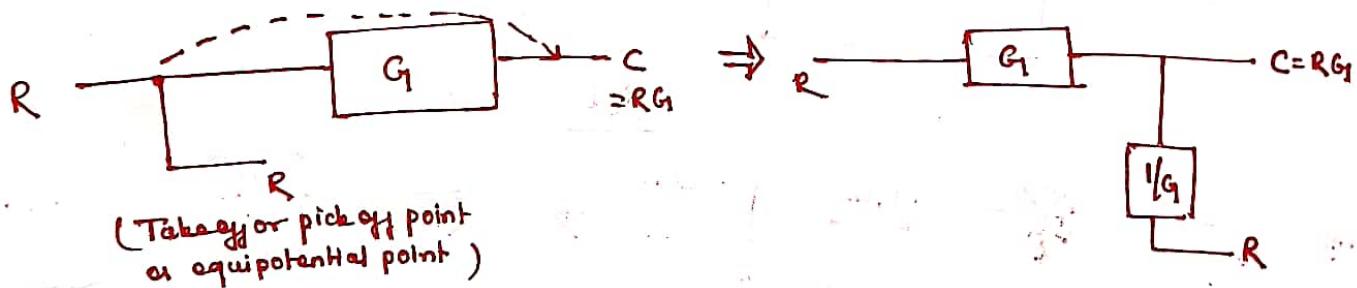
2. Blocks are Connected in Series / cascade:



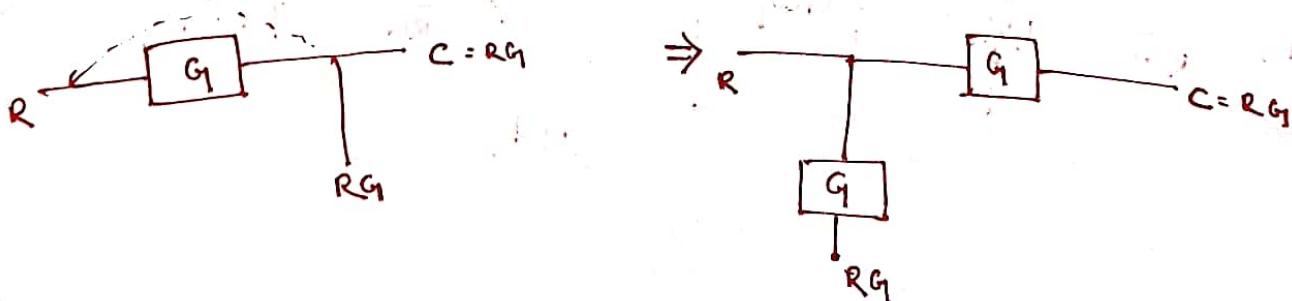
3. Blocks are Connected in parallel:



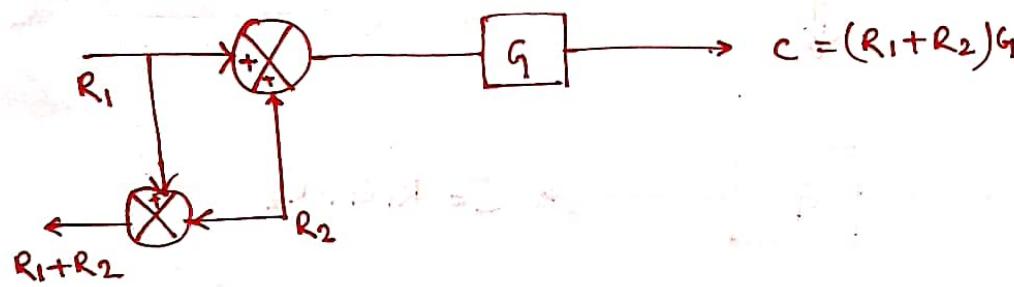
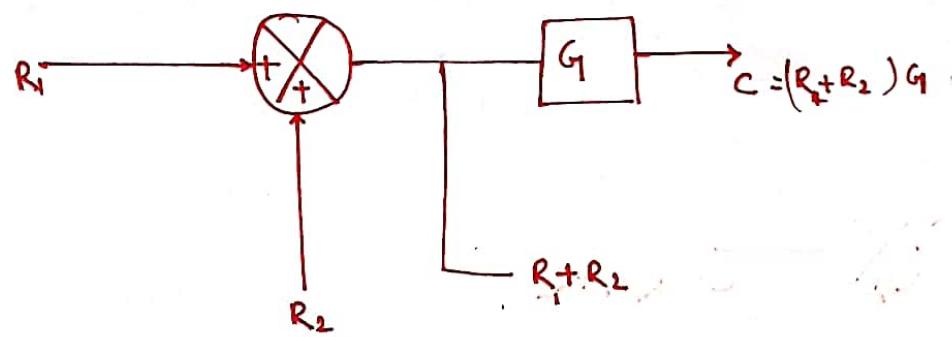
4. Shifting of a take off point after a Block:



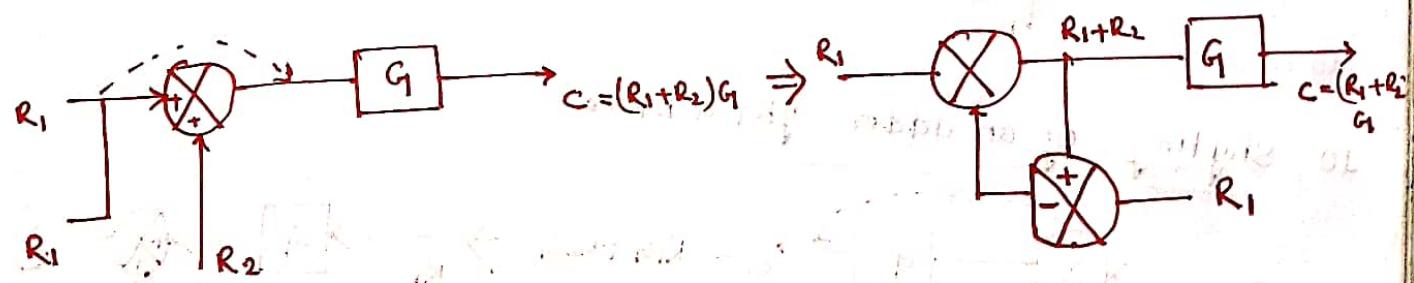
5. Shifting of a take off point before a block:



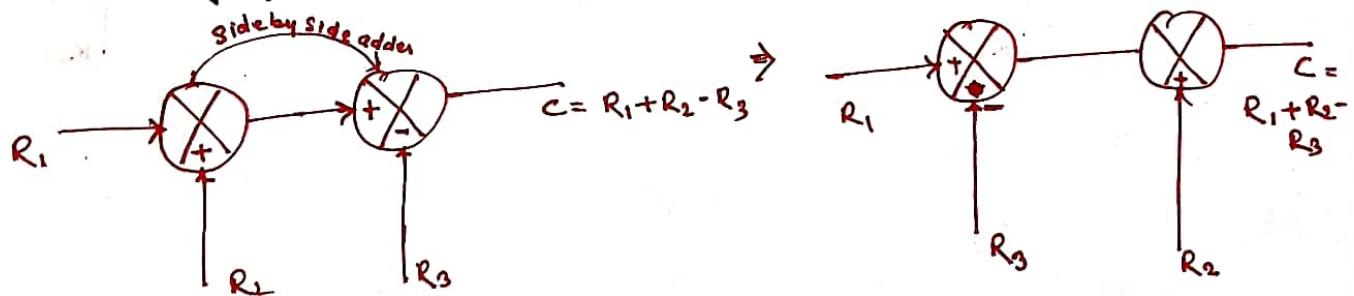
## 6. Shifting of a takeoff point before an Adder



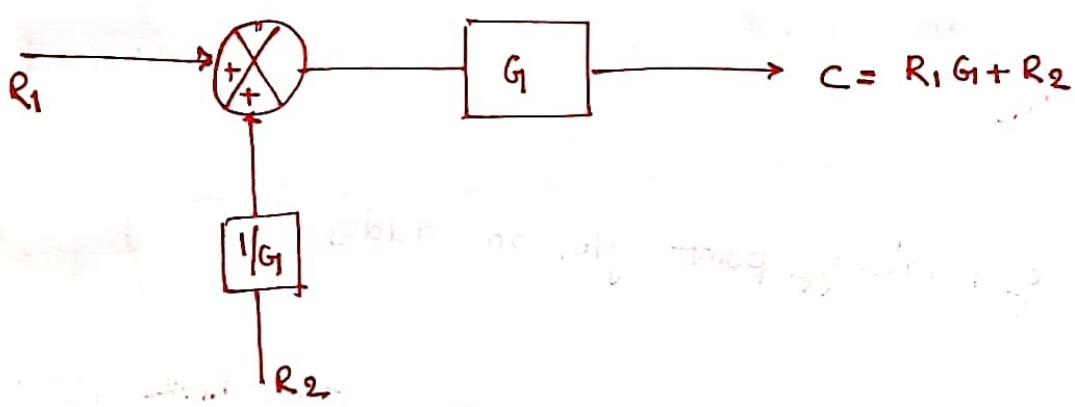
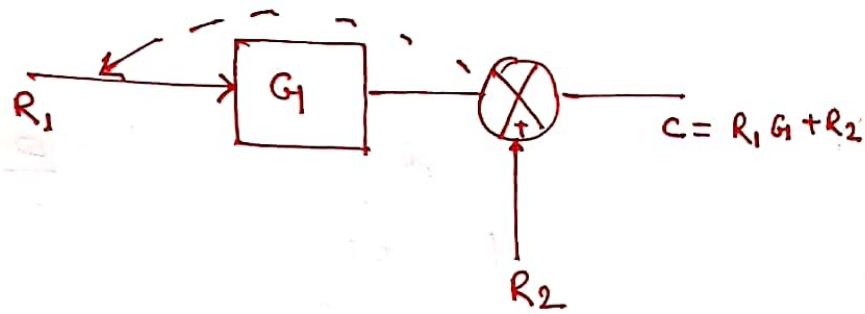
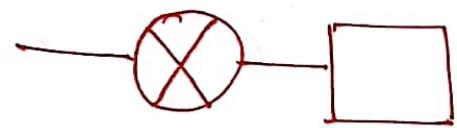
## 7. Shifting of a takeoff point after an adder



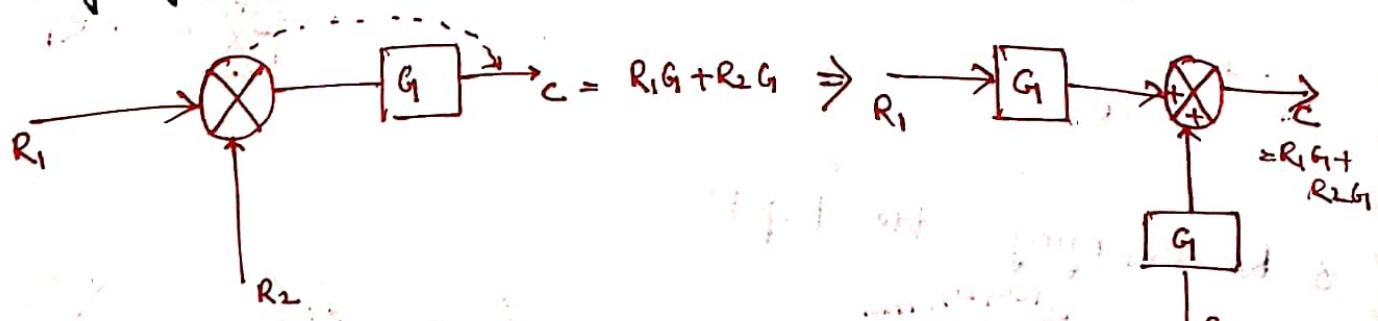
## 8. Re-arranging adder Input?



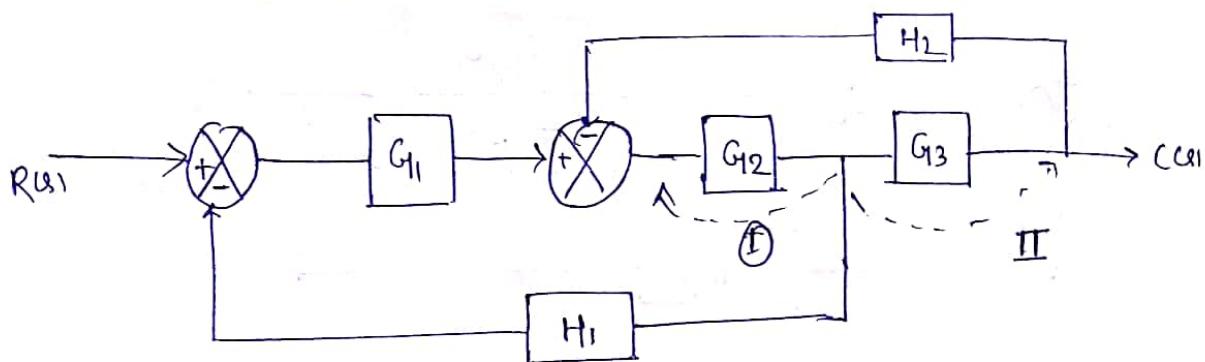
9. Shifting of an adder before a block:



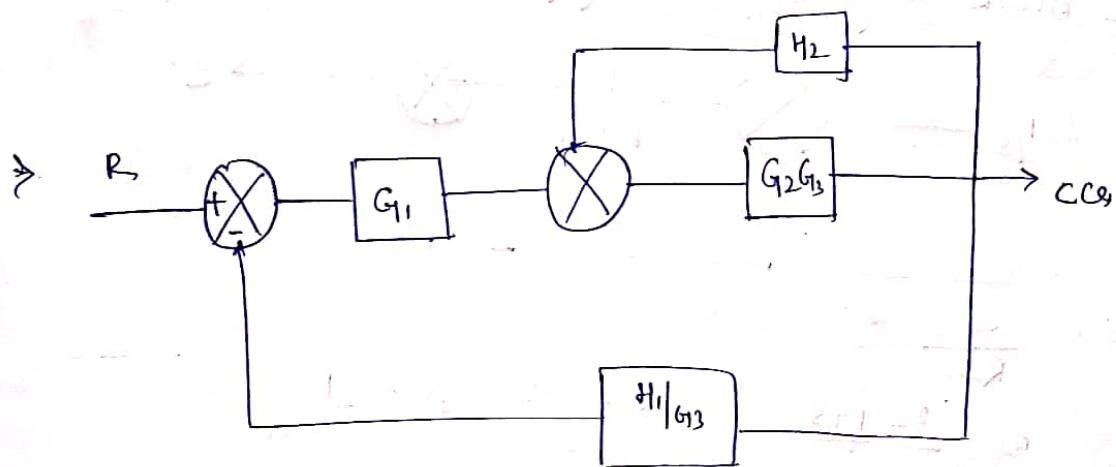
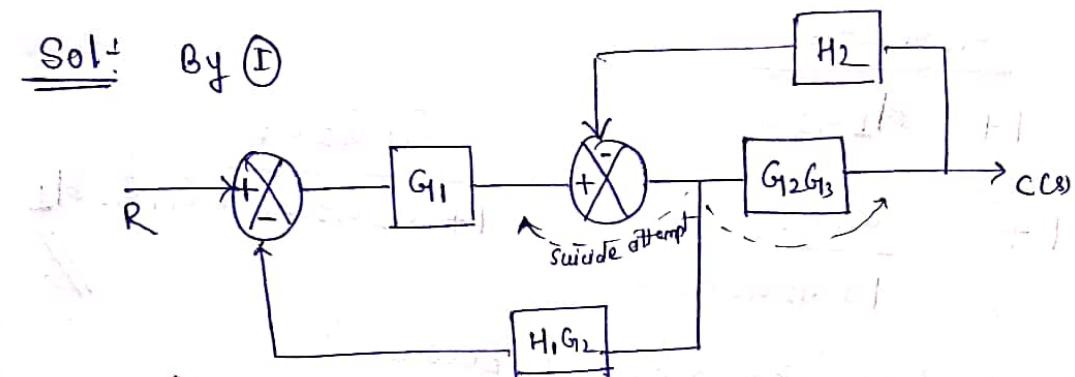
10. Shifting of an adder after a block:



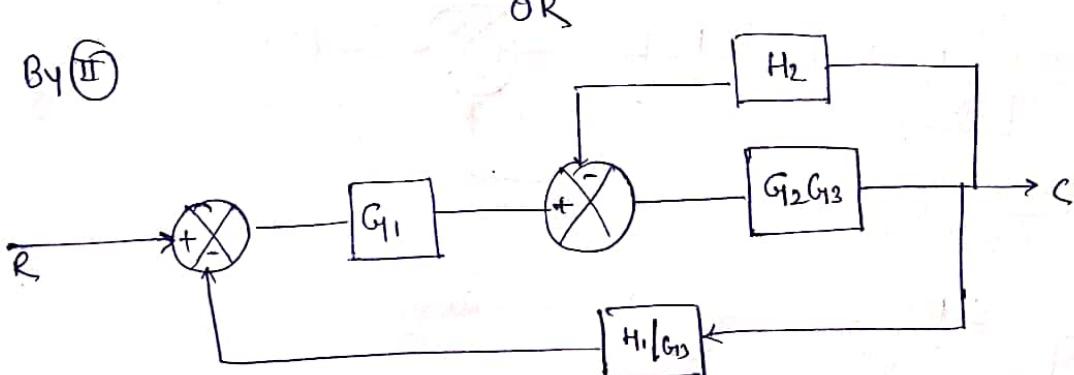
Q1. Find transfer function  $C(s)/R(s)$  for a diagram given below:



Sol: By ①



By ②



By ① 2step  
By ② 1step

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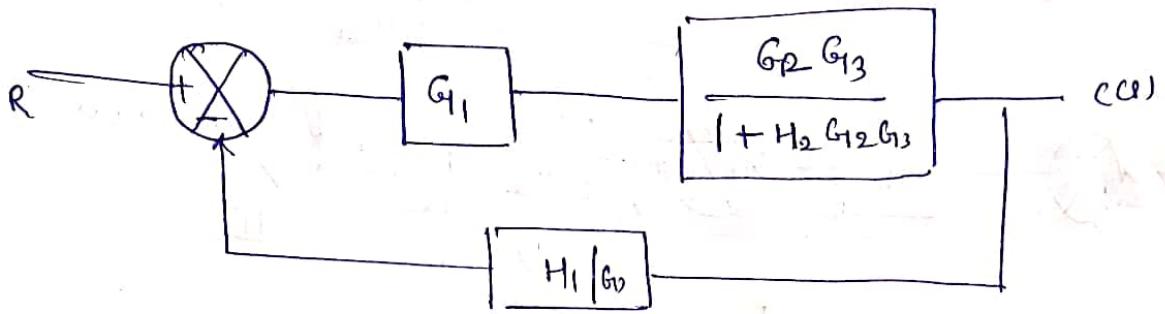
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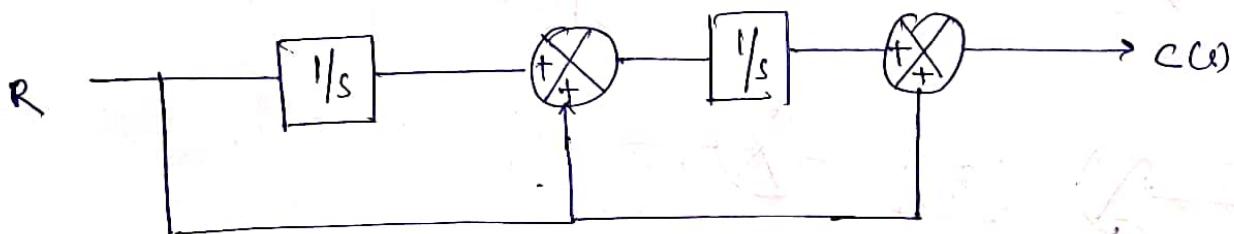
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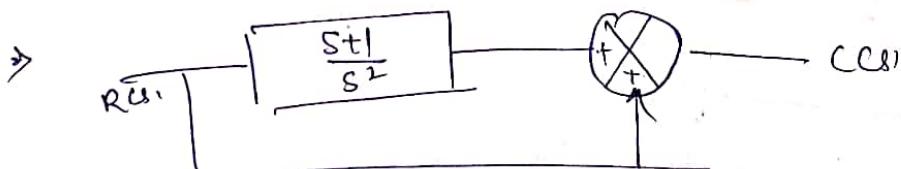
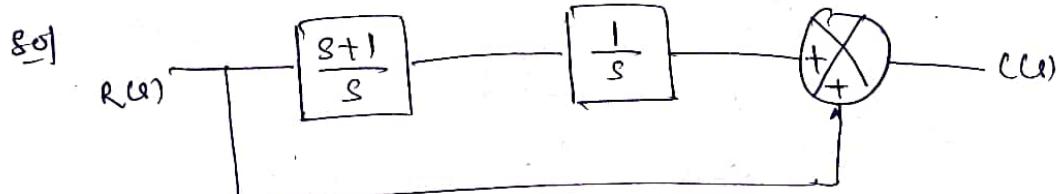
$$\frac{C}{R} = \frac{\frac{G_1 G_2 G_3}{1 + H_2 G_1 G_2 G_3}}{1 + \frac{\frac{G_1 G_2 G_3}{1 + H_2 G_1 G_2 G_3} + H_1}{1 + H_2 G_1 G_2 G_3}}$$

Q2. For the block diag. shown in fig. below



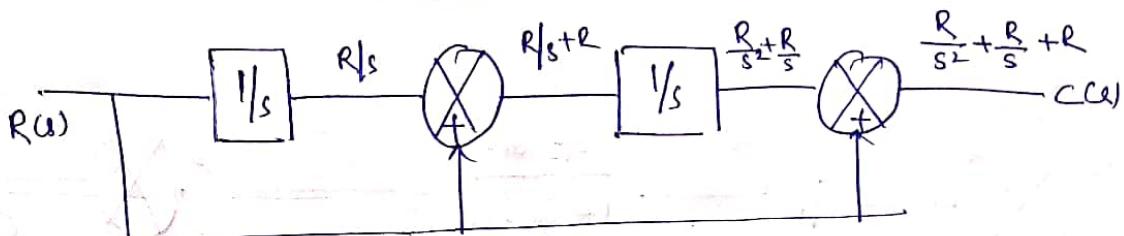
The xfer function  $\frac{C(s)}{R(s)} = ?$

- a)  $\frac{s^2+1}{s^2}$    b)  $\frac{s^2+1+s}{s^2}$    c)  $\frac{s^2+s+1}{s}$    d)  $\frac{1}{s^2+s+1}$



$$\frac{C}{R} = \frac{s+1}{s^2} + 1 \Rightarrow \frac{s+1+s^2}{s^2}$$

OR



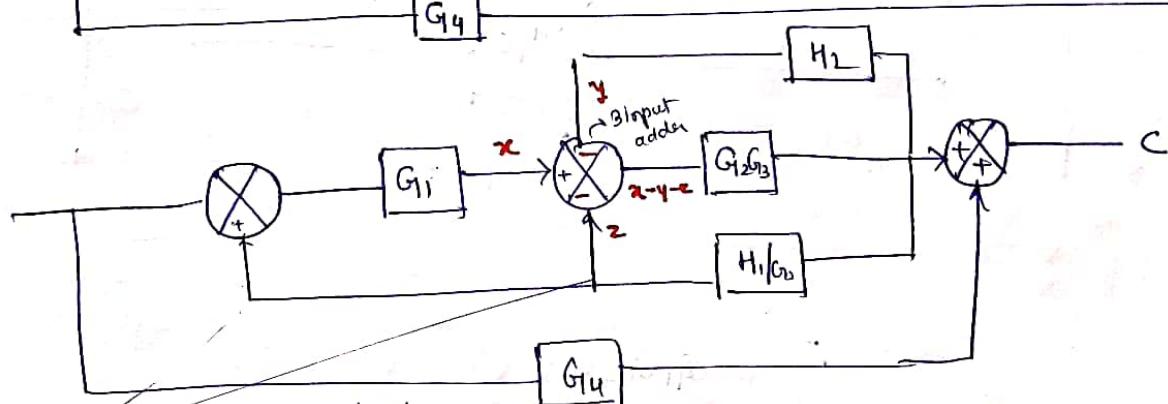
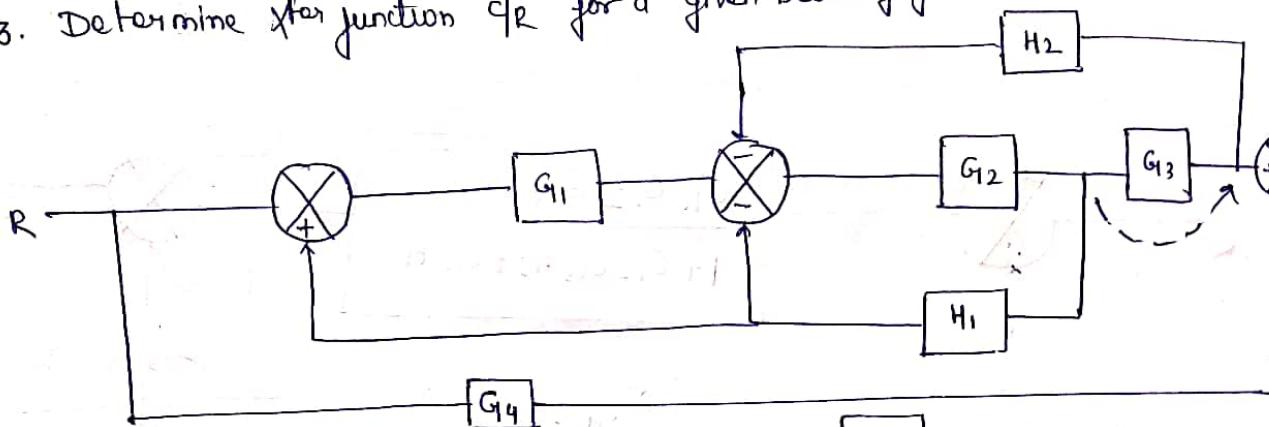
(Other good approach)

$$\Rightarrow E = \frac{R}{s^2} + \frac{R}{s} + L$$

$$\frac{C}{R} = \frac{1}{s^2} + \frac{1}{s} + 1$$

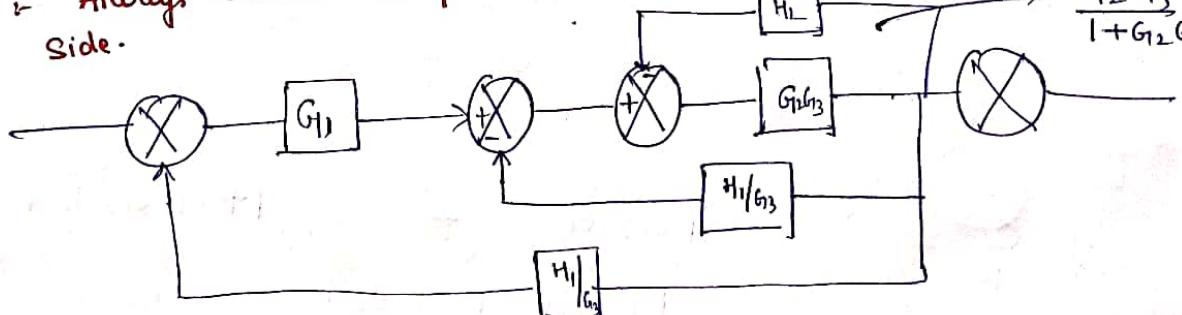
$$\frac{C}{R} = \frac{s^2 + s + 1}{s^2}$$

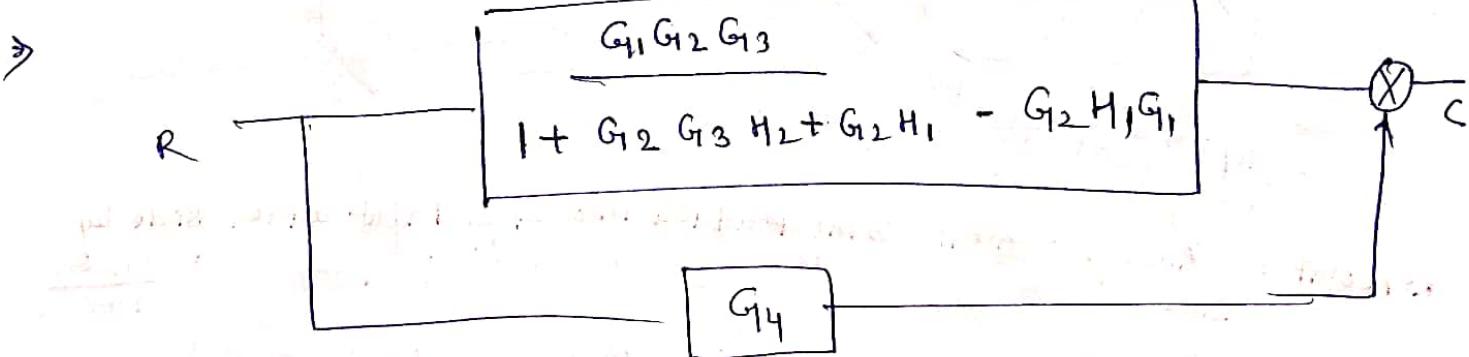
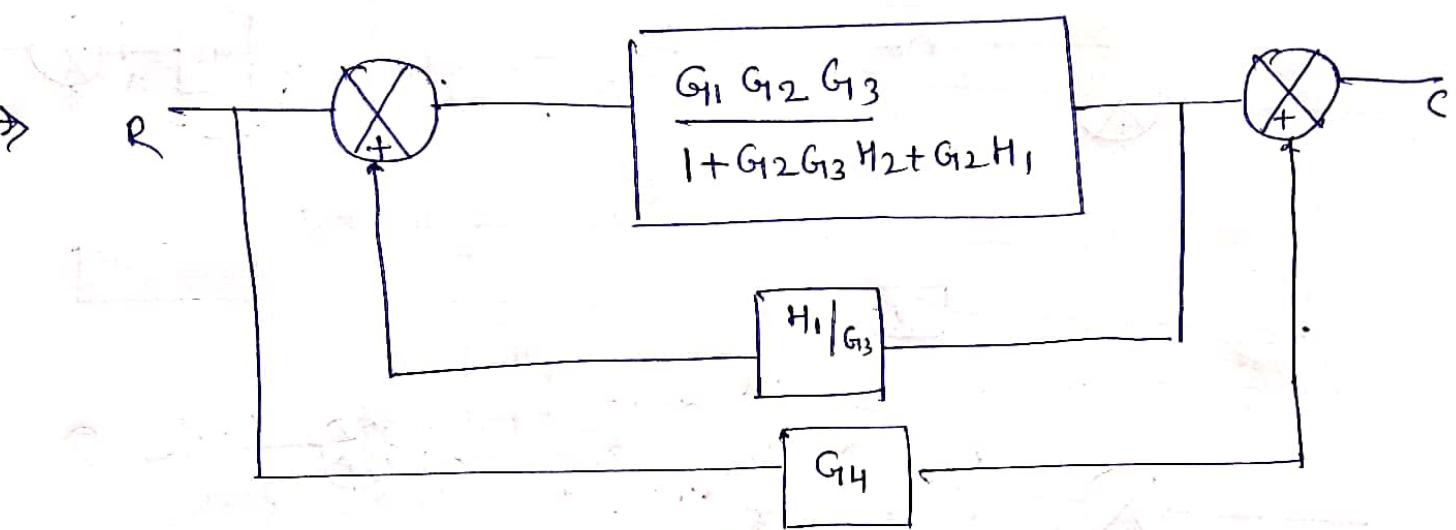
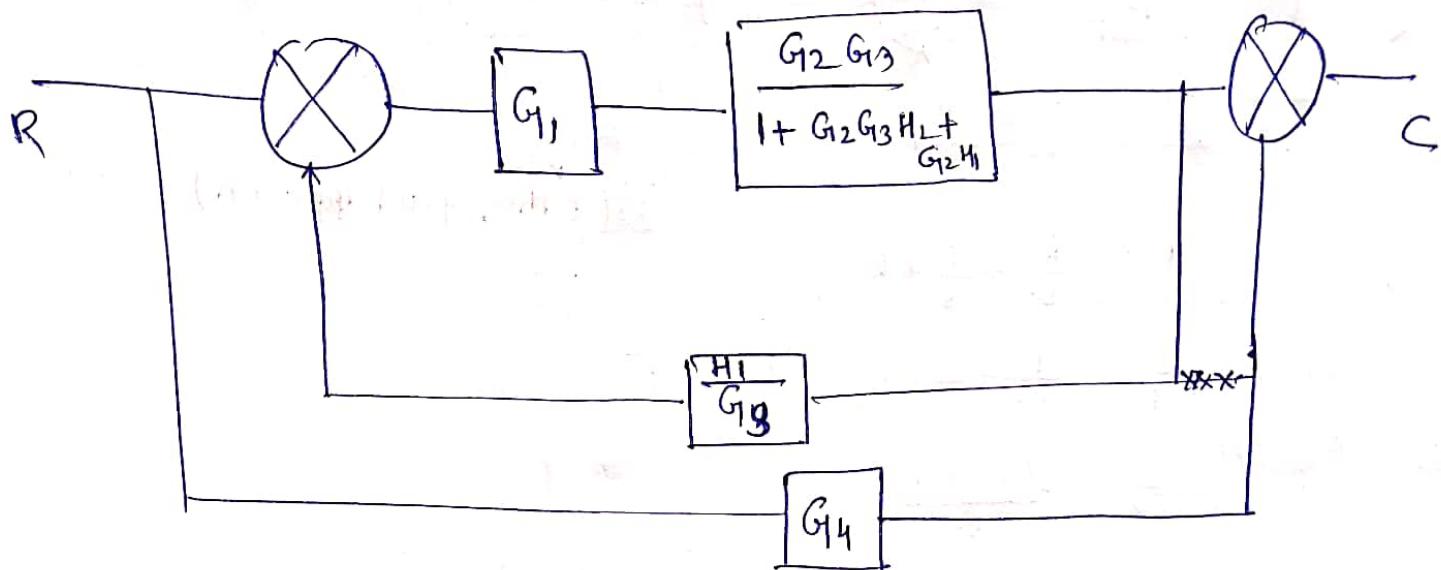
Q3. Determine  $\frac{dI}{dt}$  at junction C/R for a given below figure.



equipotential  
can be replaced by two blocks

Key point - Always convert 3 input added into 2, 2 input adder side by side.





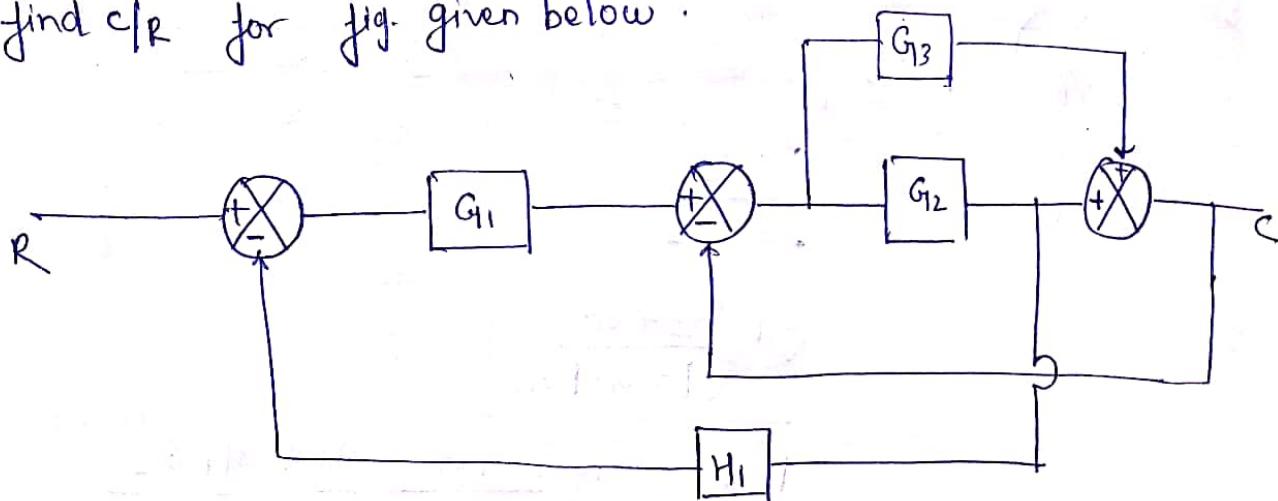
$$\frac{C}{R} = \frac{\frac{G_1 G_2 G_3}{1 + G_2 G_3 H_2 + G_2 H_1} + G_4 (1 + G_2 G_3 H_2 + G_2 H_1 - G_2 H_1 G_1)}{1 + G_2 G_3 H_2 + G_2 H_1 - G_2 H_1 G_1}$$

//

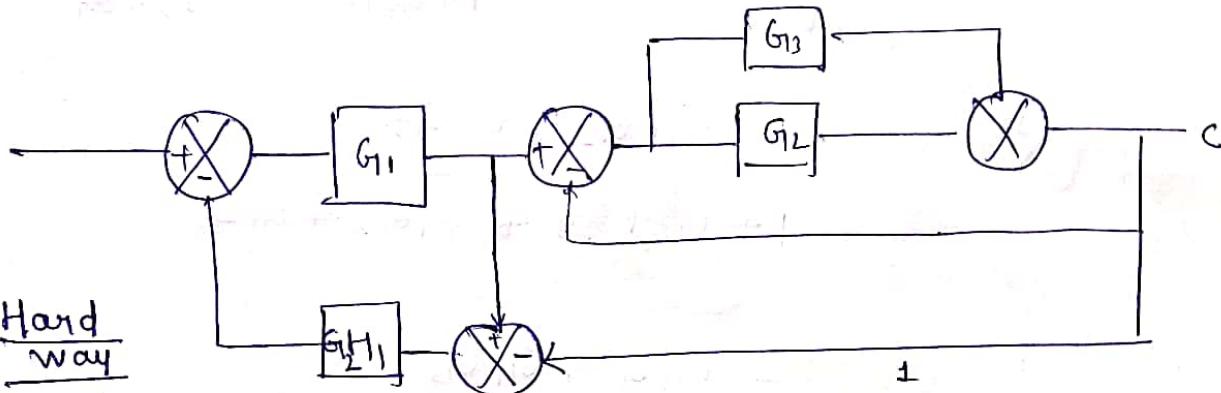
H.00

without converting 3 input adder in 2,2 input adder and check Ans.

Q. find C/R for fig. given below.

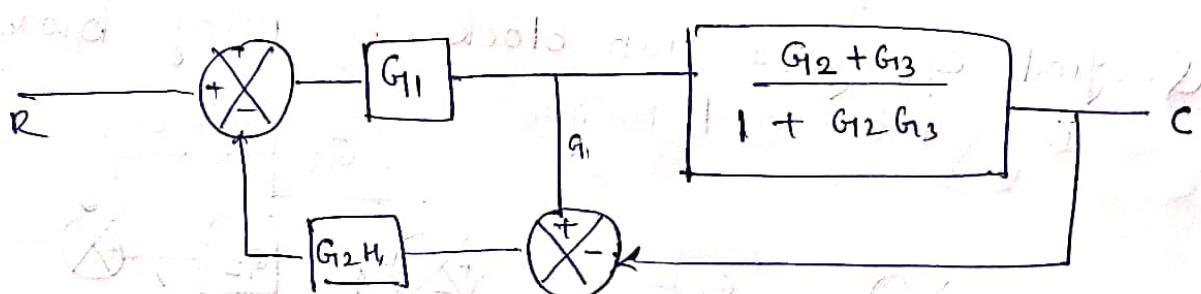


Sol.

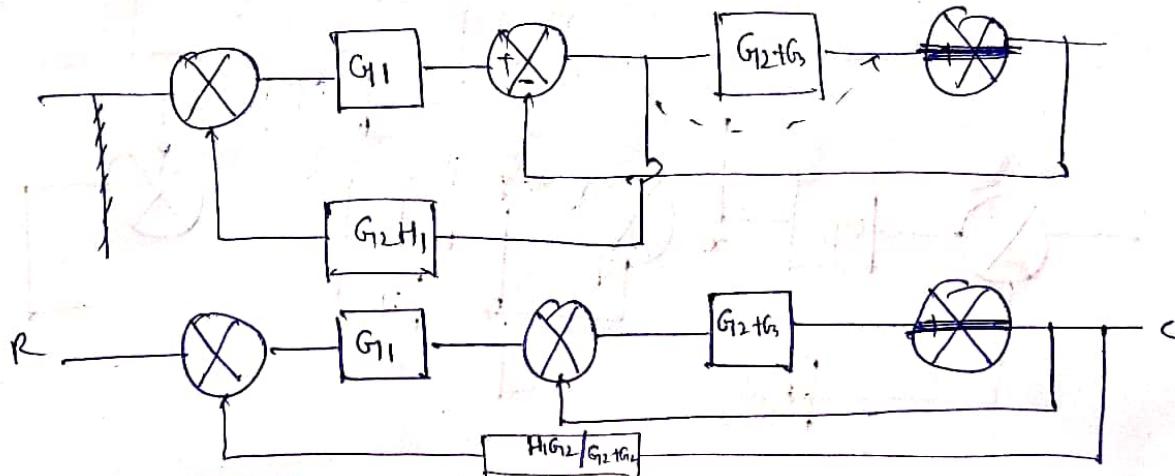


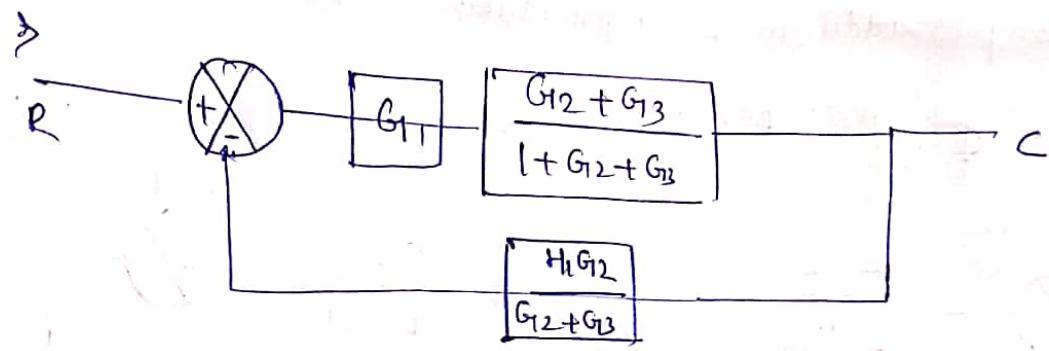
Hard way

Adder before, after



(Comp.)  
Other Method



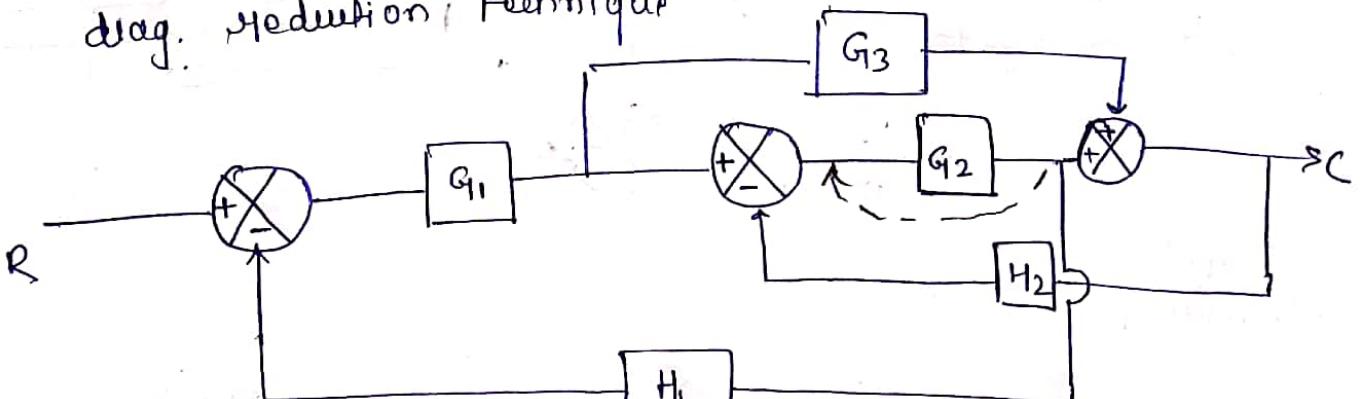


$$\frac{G_{11} \left( \frac{G_2 + G_3}{1 + G_2 + G_3} \right)}{1 + \frac{G_{11} \left( \frac{G_2 + G_3}{1 + G_2 + G_3} \right)}{1 + G_2 + G_3}} \times \frac{H_1 G_2}{G_2 + G_3}$$

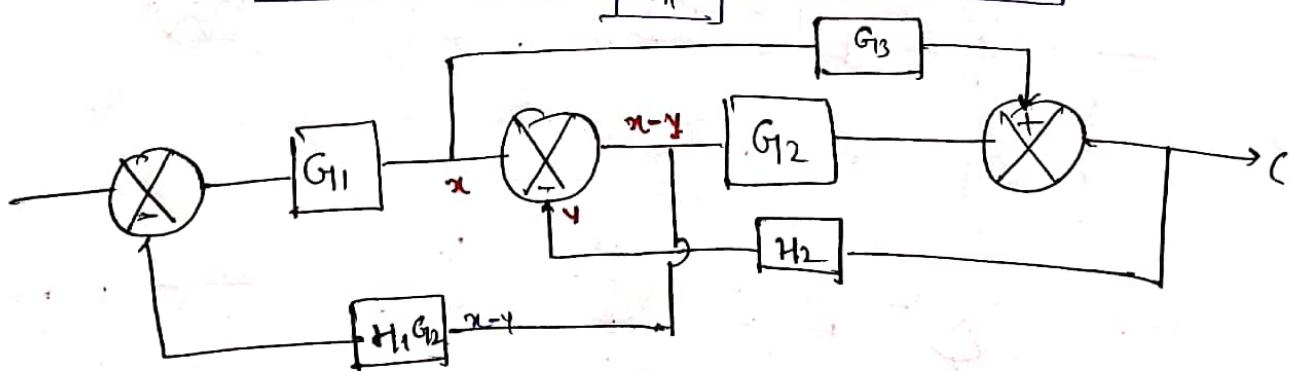
$$\frac{G_{11} G_2 + G_{11} G_3}{1 + G_2 + G_3 + G_{11} G_2 + G_{11} G_3}$$

$$\frac{G_{11} G_2 + G_{11} G_3}{1 + G_2 + G_3 + G_{11} G_2 H_1}$$

Q. find C/R for a given block by using Block diag. reduction technique



Sol

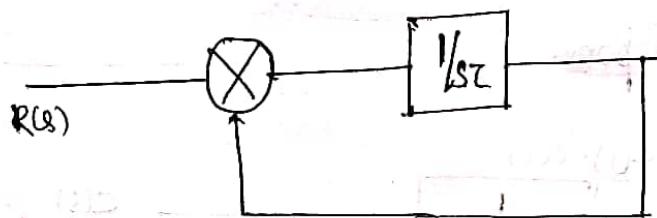


## Chapter - 3

### Time Response

1. Time Response describe the performance of the system w.r.t time.
2. Time Response is a combination of transient and steady state response.
3. Time Response gives the speed and error of the system.

#### \* Analysis of first Order System:



$$\frac{C(s)}{R(s)} = \frac{G(s)}{1 + G(s)}$$

$$T(s) = \frac{1/sT}{1 + 1/sT}$$

$$T(s) = \frac{1}{1 + sT}$$

$$\text{Bandwidth} = \frac{1}{T_c} = \frac{1}{\tau} \text{ rad/sec}$$

Stable System

Time Constant =  $\tau$

first order system

i) DC gain = 1

ii)  $\tau = 2\text{sec}$

$$T(s) = \frac{1}{1 + 2s}$$

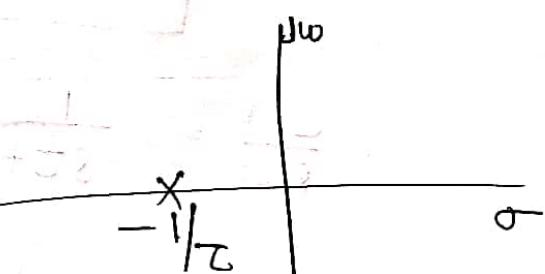
first order system

i) DC gain  $\neq 0$

ii)  $\tau = 2\text{sec}$

$$T(s) = \frac{1}{1 + 2s} X$$

$$T(s) = \frac{10}{1 + 2s}$$



$$T(s) = \frac{k e^{-\tau s}}{1+s\tau}$$

$$\text{DC gain} = k$$

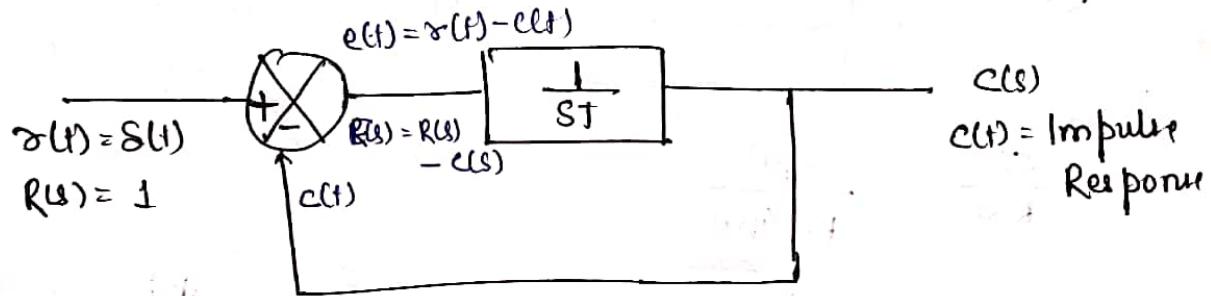
iii)  $\text{Degain} = 5$

$$\tau = 3 \text{ sec}$$

$$\text{Delay time} = 28 \text{ sec}$$

$$T(s) = \frac{5 e^{-2s}}{1+2s}$$

### Case I Unit Impulse Response



$$\frac{C(s)}{R(s)} = \frac{1}{s\tau + 1}$$

$$C(s) = \frac{R(s)}{1 + s\tau}$$

$$C(s) = \frac{1}{1 + s\tau}$$

$$C(t) = \frac{1}{1 + \frac{t}{\tau}} = \frac{1}{\tau} e^{-t/\tau} u(t)$$

$$C(t) = \frac{1}{1 + \frac{t}{\tau}}$$

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$$e(t) = \delta(t) - c(t)$$

$$E(s) = R(s) - C(s)$$

steady state error

$$e_{ss} = \lim_{t \rightarrow \infty} e(t)$$

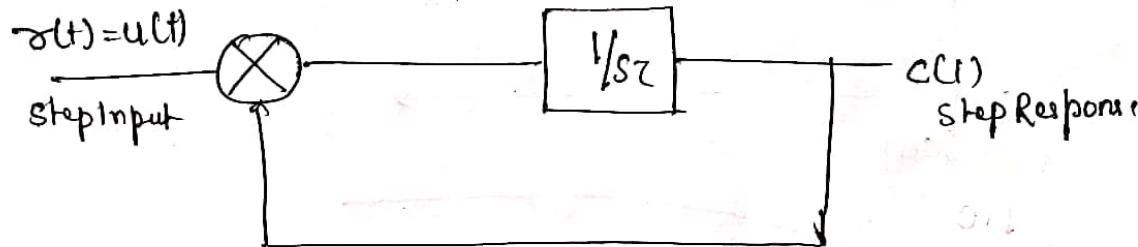
$$e_{ss} = \lim_{s \rightarrow 0} s E(s)$$

$$e(t) = R(t) - \frac{1}{\tau} e^{-t/\tau} u(t)$$

key point

- Steady state error does not exist for Impulse response because we cannot compare input & output at  $t = \infty$

## Case II Unit Step Response



$$\frac{C(s)}{R(s)} = \frac{1}{sT + 1}$$

$$C(s) = \frac{1/s}{1 + sT}$$

$$= \frac{1/s}{\frac{1}{T}(s + 1/\tau)}$$

$$= \frac{1/\tau}{s(s + 1/\tau)}$$

$$C(s) = \frac{1}{s} - \frac{1}{s+1/2}$$

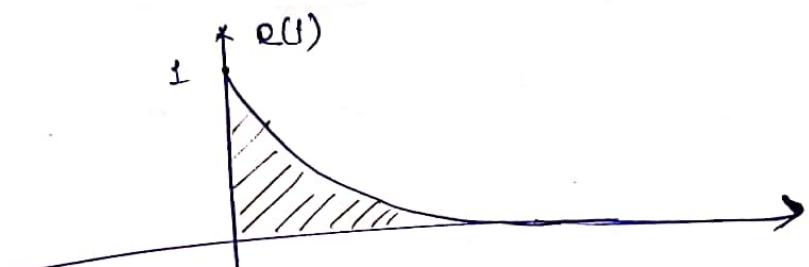
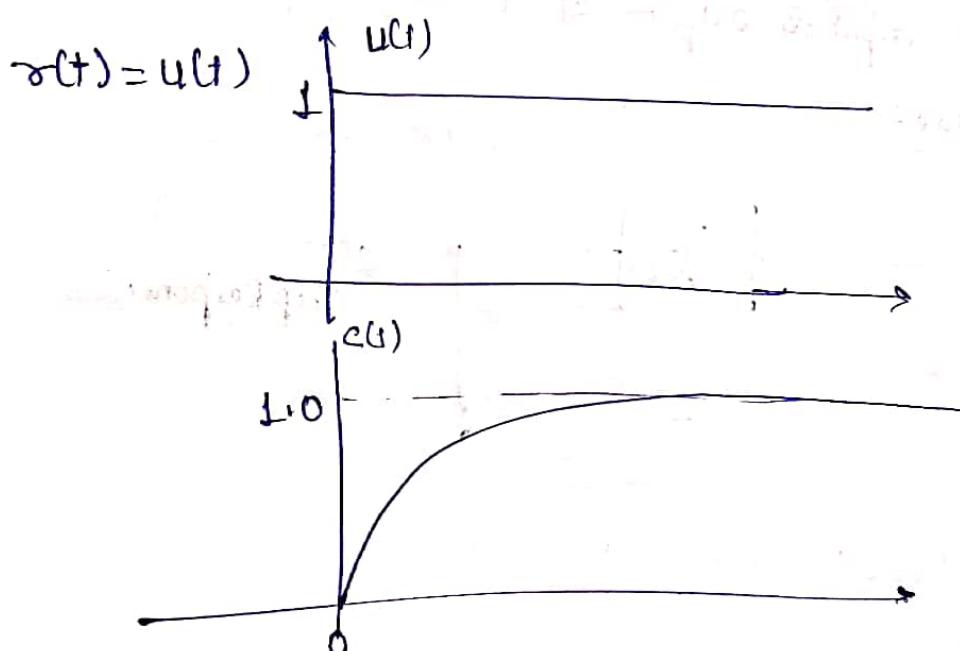
$$C(t) = 1 - e^{-t/2}$$

$$C(t) = u(t) - e^{-t/2} u(t)$$

$$C(t) = (1 - e^{-t/2}) u(t)$$

$$\text{error } e(t) = u(t) - C(t) + e^{-t/2} u(t)$$

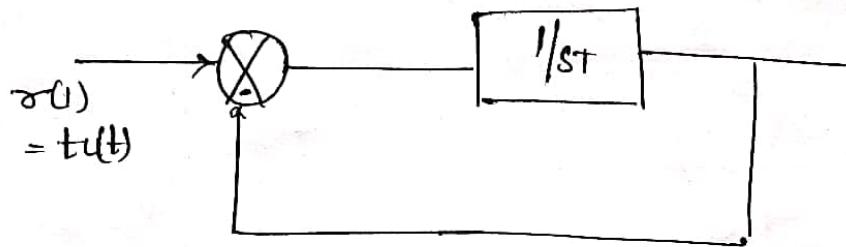
$$e(t) = e^{-t/2} u(t)$$



$$e_{ss} = \lim_{t \rightarrow \infty} e^{-t/2} u(t)$$

$$e_{ss} = 0$$

### Case III Unit Ramp Response



$c(t) \Rightarrow$  Unit ramp resp  
 $c(s) \Rightarrow$   $\frac{1}{s^2}$

$$\frac{C(s)}{R(s)} = \frac{1}{1+s\tau}$$

$$C(s) = \frac{1}{s^2(1+s\tau)}$$

$$C(s) = \frac{1}{s^2(1/2)}$$

$$\frac{1}{2} = \frac{A}{s} + \frac{B}{s^2} + \frac{C}{s+1/2}$$

$$s=0$$

$$\frac{1}{2} = A\left(s+\frac{1}{2}\right)s + B(s+1/2) + Cs^2$$

$$\frac{1}{2} = B\left(\frac{1}{2}\right)$$

$$B = 1$$

$$s = -1/2$$

$$\frac{1}{2} = C \frac{1}{s^2}$$

$$C = 2$$

$S^2$  compare

$A + c \geq 0$

$$A = -c$$

$$N = -\tau$$

$$C(t) = \frac{-\tau}{s} + \frac{1}{s^2} + \frac{\tau}{s+1/\tau}$$

$$C(t) = -\tau u(t) + t u(t) + \tau e^{-t/\tau} u(t)$$

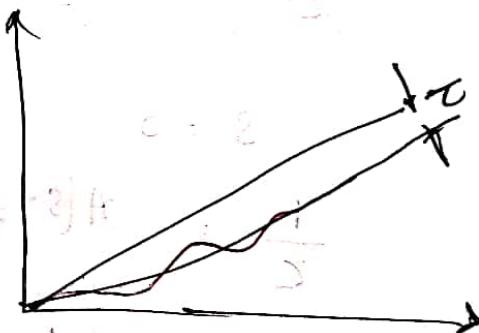
$$e(t) = \gamma(t) - C(t)$$

$$= t u(t) + \tau(u(t)) - t u(t) - \tau e^{-t/\tau} u(t)$$

$$e(t) = \tau u(t) - \tau e^{-t/\tau} u(t)$$

$$e_{ss} = \lim_{t \rightarrow \infty} e(t) = \lim_{t \rightarrow \infty} \tau u(t) - \tau e^{-t/\tau} u(t) \\ = \tau u(\infty)$$

$$e(\infty) = e_{ss} = \tau$$



Case IV parabolic input

$$R(s) = 1/s^3 \quad \gamma(t) = \frac{t^2}{2}$$

$$\frac{C(s)}{R(s)} = \frac{1}{s^2 + 1}$$

$$C(s) = \frac{1}{s^3(s^2 + 1)}$$

## Chapter-4 Routh Hurwitz Stability

- Routh Hurwitz describes the stability of control system.
- Stability is defined in two parameters—
  - i) Absolute stability
  - ii) Relative stability.
- In Case of Absolute stability, It is defined in terms of location of poles & for CLS to be stable all pole should lie on left hand side of S-plane.

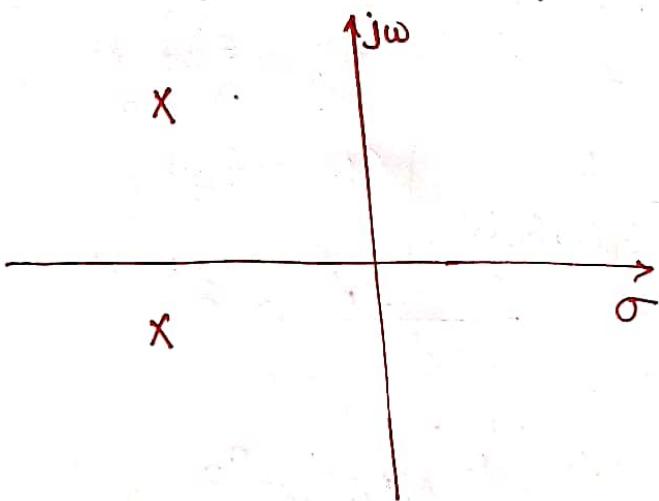


fig a

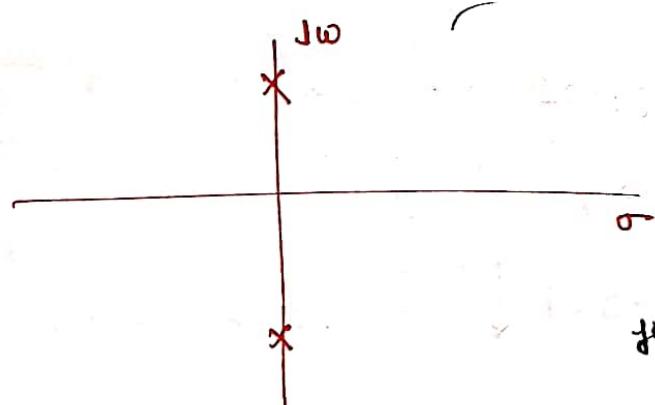
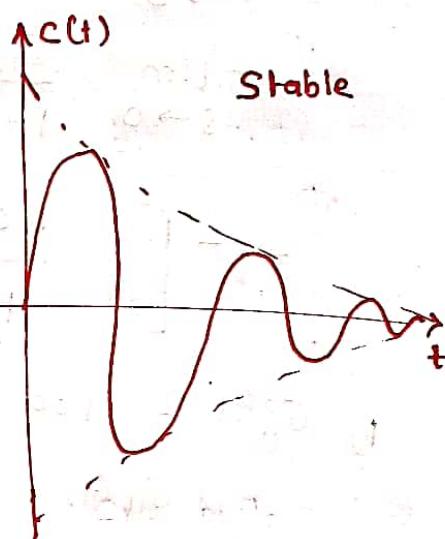


fig b

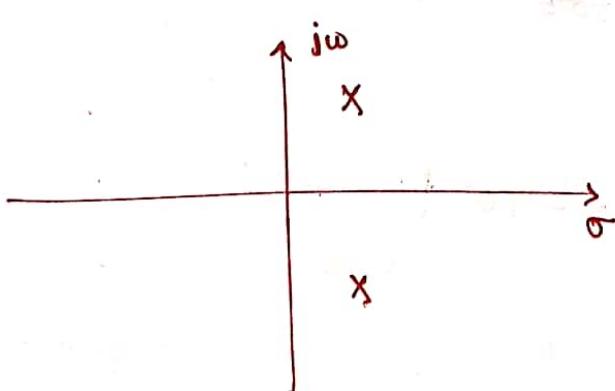
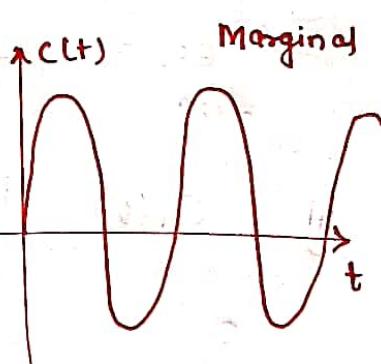
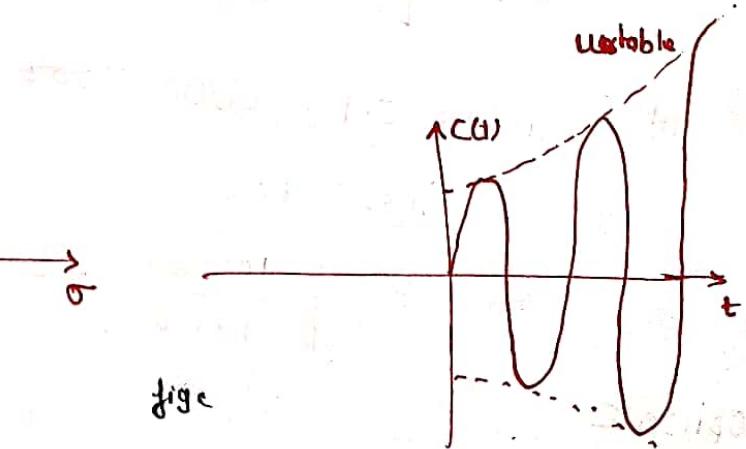


fig c



- In Case of Relative stability it is defined in terms of damping ration, Gain Margin, phase Margin.

## The Properties of R-H Stability Criterion:

- The R-H Stability Criterion is applicable for closed loop system because formation of Routh table we use characteristic eqn which is defined for closed loop system.
- Key point :- It is also applicable for determining stability of OLTF or stability of any polynomial.
- For closed loop system to be stable, all elements of first column in Routh table must be same sign (+ve/-ve).
- Number of Sign changes in first Column represent the numbers of poles in RHS of S plane. If System will become Unstable.
- If any power of 's' is missing in characteristic eqn that represents the presence of at least one pole in the RHS of S plane & System will become Unstable.
- For R-H Stability, Coefficient of characteristic equation must be real that means neither Imaginary nor Complex nor Sinusoidal.
- If characteristic eqn contain only even power of s that represent presence of pole on Imaginary axis. & System become Marginal stable.
- If Imaginary axis contain repeated poles then System will become Unstable.
- Routh table give information of number of Right-hand pole in the S plane.

# Formation of Routh Table

Consider characteristic eqn -

$$1 + G(s)H(s) = a_0 s^6 + a_1 s^5 + a_2 s^4 + a_3 s^3 + a_4 s^2 + a_5 s + a_6 = 0$$

Given - Rows

$$\begin{array}{cccccc} s^6 & a_0 & a_2 & a_4 & a_6 \\ s^5 & a_1 & a_3 & a_5 & 0 \\ s^4 & b_1 & b_2 & b_3 & 0 & b_1 = \frac{a_1 a_2 - a_0 a_3}{a_1} \\ s^3 & c_1 & c_2 & c_3 & 0 & b_2 = \frac{a_1 a_4 - a_0 a_5}{a_1} \\ s^2 & d_1 & d_2 & 0 & 0 & b_3 = \frac{a_1 a_6 - a_0 a_0}{a_1} \\ s^1 & e_1 & e_2 & 0 & 0 & c_1 = \frac{d_1 c_2 - c_1 d_2}{d_1} \\ s^0 & f_1 & f_2 & 0 & 0 & c_2 = \frac{b_1 a_5 - a_1 b_3}{b_1} \end{array}$$

$$\begin{aligned} \text{eq1 } P(s) &= s^3 + 2s^2 + 8s + 10 \\ \text{eq2 } P(s) &= s^3 + 2s^2 + 8s + 26 \\ \text{eq3 } P(s) &= s^3 + 2s^2 + 8s + 16 \end{aligned}$$

$$\begin{aligned} \text{Sol 1) } P(s) &= s^3 + 2s^2 + 8s + 10 \\ s^3 + 1 & 8 \\ s^2 + 2s + 10 & \\ s^1 + 3 & \\ s^0 + 10 & \end{aligned}$$

$s^3$	1	no sign change
$s^2$	2	no sign change
$s^1$	3	no sign change
$s^0$	10	no sign change

i) Number of sign changes in first column = 0

ii) Number of Right hand poles = 0

iii) Number of left hand poles = 3

iv) Number of Imaginary axis poles = 0

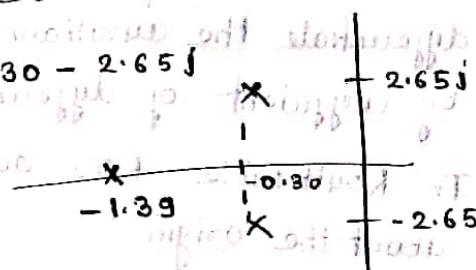
v)  $P(s)$  = Stable

$$s^3 + 2s^2 + 8s + 10 = 0$$

$$s = -1.39, -0.30 + 2.65j, -0.30 - 2.65j$$

poles formed are right half plane

{using calculator}



eg 2  $P(s) = s^3 + 2s^2 + 8s + 20$

<u>Sol</u>	$s^3$	1	8
	$s^2$	2	20
	$s$	-2	
	$s^0$	20	

$s^3 + 1$	No
$s^2$	2
$s$	-2
$s^0$	20

i) Number of sign changes in first column = 2

ii) Number of Right hand pole = 2

iii) Number of left hand pole = 1

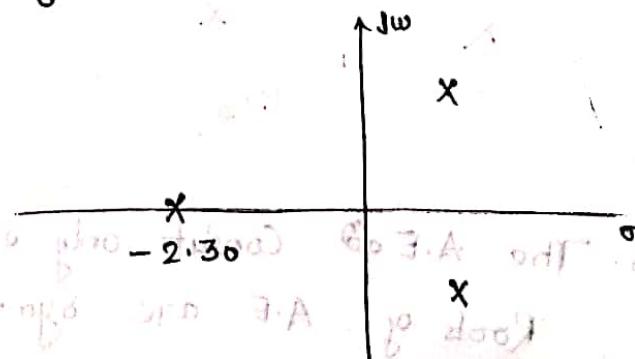
iv) Number of Imaginary pole = 0

v)  $P(s)$  = Unstable

$$s^3 + 2s^2 + 8s + 20 = 0$$

$$s = -2.30, 0.18 + j2.94$$

$$0.18 - j2.94$$



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Eq 3

$$P(s) = s^3 + 2s^2 + 8s + 16$$

Sol

$$s^3 \quad 1 \quad 8$$

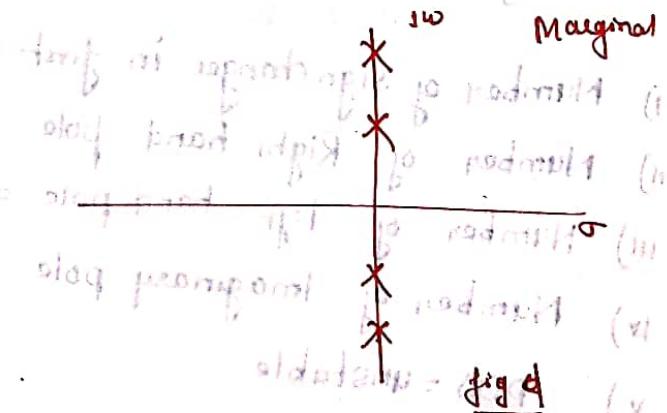
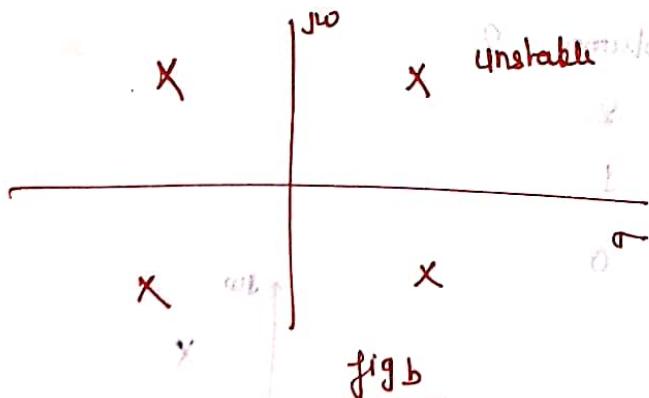
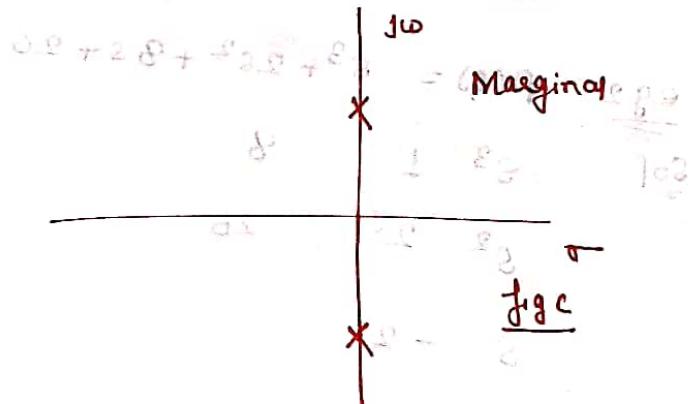
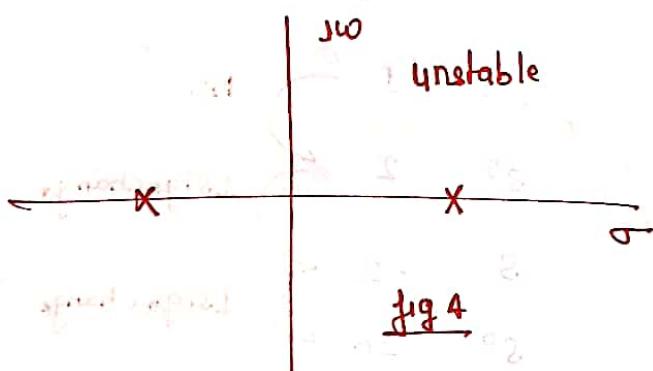
$$s^2 \quad 2 \quad 16$$

$s^1$	0	0
$s^0$		

(Row of zeros) ROZ

### Row of Zero (ROZ)

- Whenever in Routh table the entire row become zero, then we require to form Auxiliary eqn and next differentiate the auxiliary equation and replace the '0' by coefficient of differential Auxiliary Eqn.
- In Routh table ROZ occur when poles are located symmetrical about the origin.



- The A.Eq<sup>n</sup> consist only even power of 's' because Roots of A.E are symmetrical about origin.

4. The Roots of  $A \cdot E_{eqn}$  are poles of closed loop system (Normally we consider closed loop system in control system).
- \* 5. The ROZ occurs only odd power of  $s$ .
6. Whenever in Routh table only 1 ROZ occur & all the Coefficient in first column are positive or same sign Then System is marginal stable because the poles are on Imaginary axis which are not repeated. (Conditional)
7. If Imaginary axis contain repeated poles then System will be Unstable.
- \* 8. If more than one ROZ occur that represents presence of repeated poles & No. of Repeated poles = No. of ROZ Undamped Natural frequency
9. Imaginary axis poles is related with Undamped Natural frequency
10. Routh table gives the exact location of Imaginary axis poles.

$$Eq. 3 \quad p(s) = s^3 + 2s^2 + 8s + 16$$

$s^3$	1	8	$\leftarrow$ 10-03
$s^2$	2	<del>16</del>	$\leftarrow$ $A \cdot E_{eqn} \quad s = -3$
$s^1$	0		$\leftarrow$ ROZ
$s^0$			

$$A(s) = 2s^2 + 16$$

$$\frac{dA(s)}{ds} = 4s + 0 = 4s$$

Replace  $\frac{dA(s)}{ds}$  Coeff with ROZ

$$\text{Result } \frac{d}{ds} \quad 0s + 0s + 1$$

$$s^3 \quad 1 \quad 8$$

$$\phi = 1 + \frac{1}{2}$$

$$\text{Polynomial } \frac{d}{ds} \quad s^2 - 8s + 16$$

$$s^2 + 2s + 8s + 16$$

$$s^2 \quad 2s - 8s + 16$$

$$s^2 + 2s + 8s + 16$$

$$s^1 \quad 4 \quad 0.$$

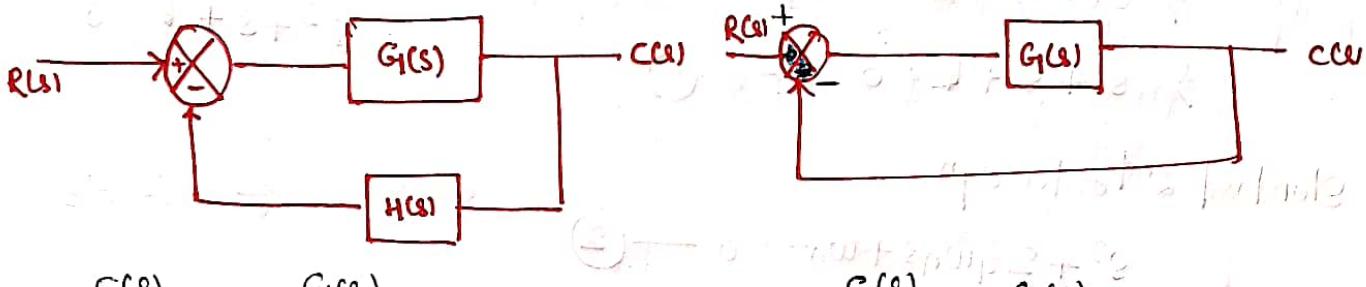
$$s^1 + 2s + 8s + 16$$

$$s^0 \quad 16 \quad 0$$

## Chapter - 5

### Root Locus

- In Root locus, Root represents poles of closed loop  $X_{fer}$  function & locus represents path of closed loop poles.
- Root locus describes the path of poles of closed loop  $X_{fer}$  function for different values of Gain of open loop  $X_{fer}$  function ( $k$ ).
- Root locus is valid for negative feedback systems.



$$\frac{C(s)}{R(s)} = \frac{G(s)}{1 + G(s)H(s)}$$

$$\frac{C(s)}{R(s)} = \frac{G(s)}{1 + G(s)H(s)}$$

char. eqn  $1 + G(s)H(s) = 0$

char. eqn  $1 + G(s) = 0$

- for negative feedback system Range of  $k$  is  $0 \text{ to } \infty$  i.e.  $0 < k < \infty$
- Inverse Root locus or Complementary Root locus is valid for positive feedback system.

Positive feedback System :-

closed loop  $\frac{C(s)}{R(s)} = \frac{G(s)}{1 - G(s)H(s)}$

Non unity feed back

char. eqn  $1 - G(s)H(s) = 0$

$$\frac{C(s)}{R(s)} = \frac{G(s)}{1 - G(s)H(s)}$$

Unity positive feed back

char. eqn  $1 - G(s) = 0$

- for positive feedback system range of  $k$  :-  $-\infty < k < 0$ .

- MATLAB Software is used for sketching the Root locus. Before MATLAB, SPURGE Instrument is used for sketching Root loci.

$$\text{eq. } G(s)H(s) = \frac{k}{s(s+1)} \quad \text{or} \quad G(s) = \frac{k}{s(s+1)}$$

$$T(s) = \frac{G(s)}{1 + G(s)H(s)} = \frac{G(s)}{1 + k} = \frac{1}{s+1}$$

$$C.E = 1 + G_1 H = 0 \\ = 1 + \frac{k}{s(s+1)} = 0 \\ s(s+1) + k = 0$$

$$\Rightarrow s(s+1) + k = 0$$

$$\Rightarrow s^2 + s + k = 0 \quad \text{--- (1)}$$

$$= s^2 + s + k = 0$$

$$= s^2 + s + k = 0$$

Standard 2nd order eqn

$$s^2 + 2\zeta\omega_n s + \omega_n^2 = 0 \quad \text{--- (2)}$$

On Comparison (1) & (2)

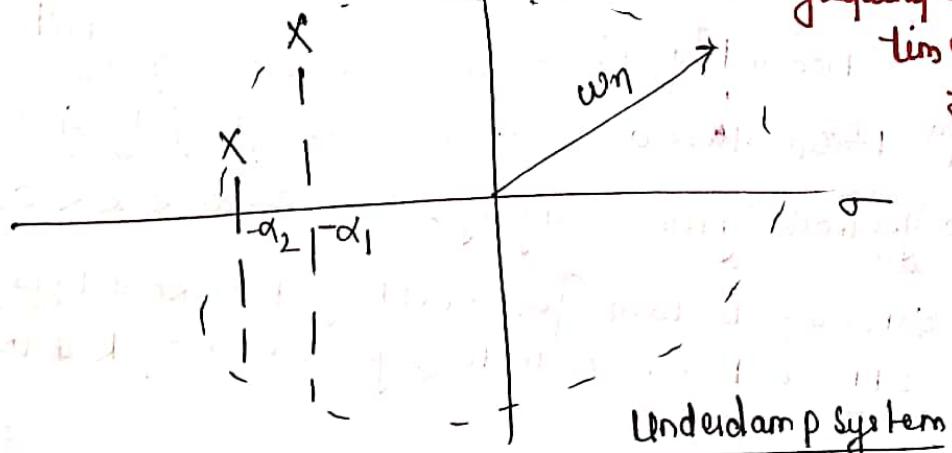
$$2\zeta\omega_n = 1 \quad \& \quad \omega_n^2 = k$$

$$\zeta = \frac{1}{2\sqrt{k}} = \text{Variable} \quad \omega_n = \sqrt{k} = \text{Variable}$$

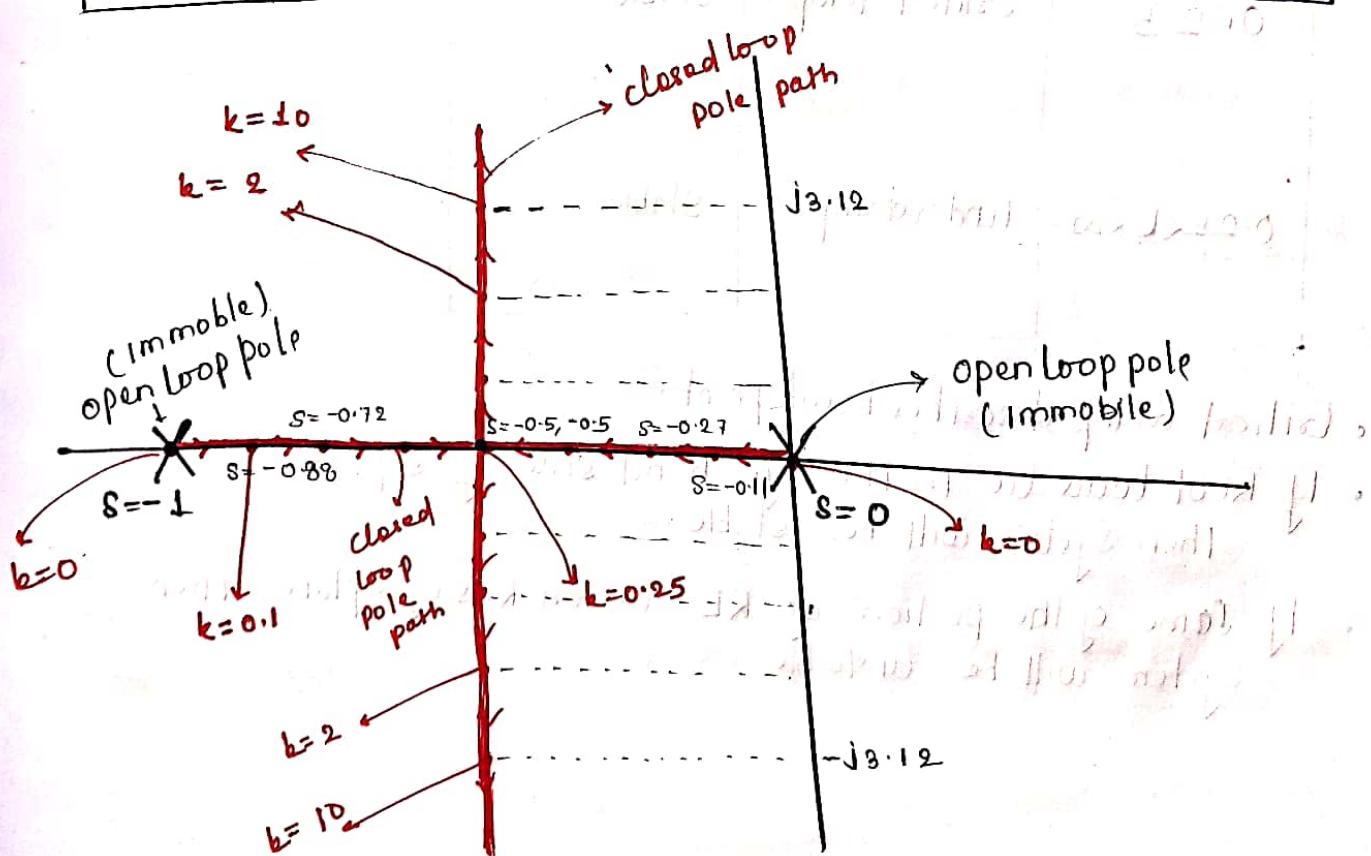
$$\Rightarrow \zeta\omega_n = \frac{1}{2\sqrt{k}} \sqrt{k} = \frac{1}{2} = \text{Constant}$$

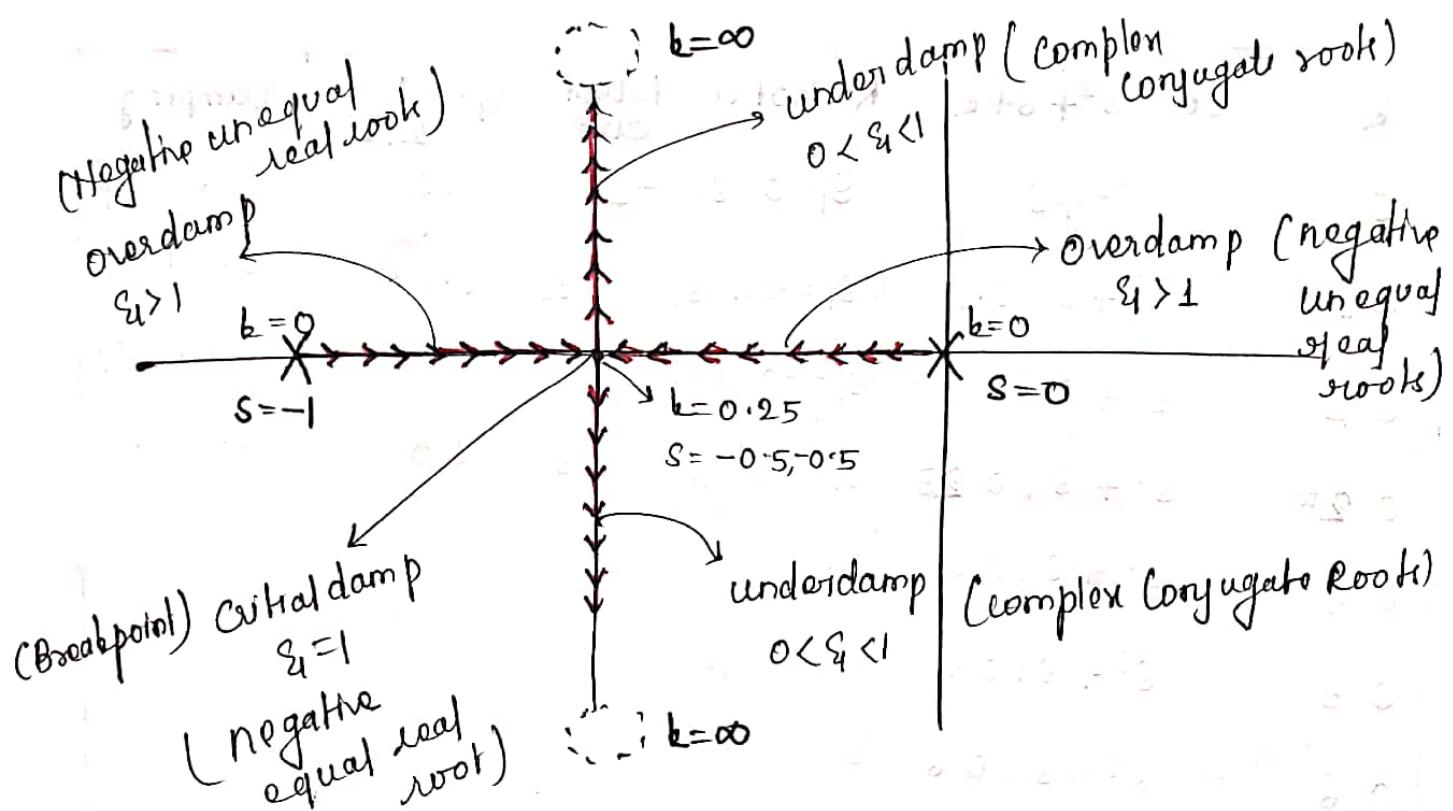
Hence.  $T = 2$  Sec

key point Root locus is based on frequency domain Analysis because frequency is Variable & time constant is fixed.



$k$	$C\dot{e} = s^2 + s + k$	Roots of $C\dot{e} = \text{Pole of cut}$	$\zeta = \frac{1}{2\sqrt{k}}$	Damping
0	$s^2 + s$	$s_1 = 0, s_2 = -1$	$\infty$	Overshoot
0.1	$s^2 + s + 0.1$	$s_1 = -0.11, s_2 = -0.88$	1.58	Overshoot
0.2	$s^2 + s + 0.2$	$s_1 = -0.27, s_2 = -0.723$	1.14	Overshoot
0.25	$s^2 + s + 0.25$	$s_1 = -0.5, s_2 = -0.5$	1.0	Critical
0.4	$s^2 + s + 0.4$	$s_1 = -0.2, s_2 = -0.8$	0.707	Underdamped
0.5	$s^2 + s + 0.5$	$s_1 = -0.25, s_2 = -0.75$	0.632	Underdamped
0.6	$s^2 + s + 0.6$	$s_1 = -0.3, s_2 = -0.6$	0.577	Underdamped
1.0	$s^2 + s + 1$	$s_1 = -0.5, s_2 = -0.5$	0.316	Underdamped
2.0	$s^2 + s + 2$	$s_1 = -0.5, s_2 = -1.5$	0.458	Underdamped
10.0	$s^2 + s + 10$	$s_1 = -0.5, s_2 = -0.5 - j3.12$	0.316	Underdamped





$k$	Damping	Stability
0	overdamp	stable
0.25	critical damp	stable
$0.25 < k < \infty$	underdamp	stable

- Critical damp describes break point.
- If Root locus lie in the left hand side of s plane then System will be stable.
- If some of the portion of RL lie on RHS of s plane then System will be unstable.

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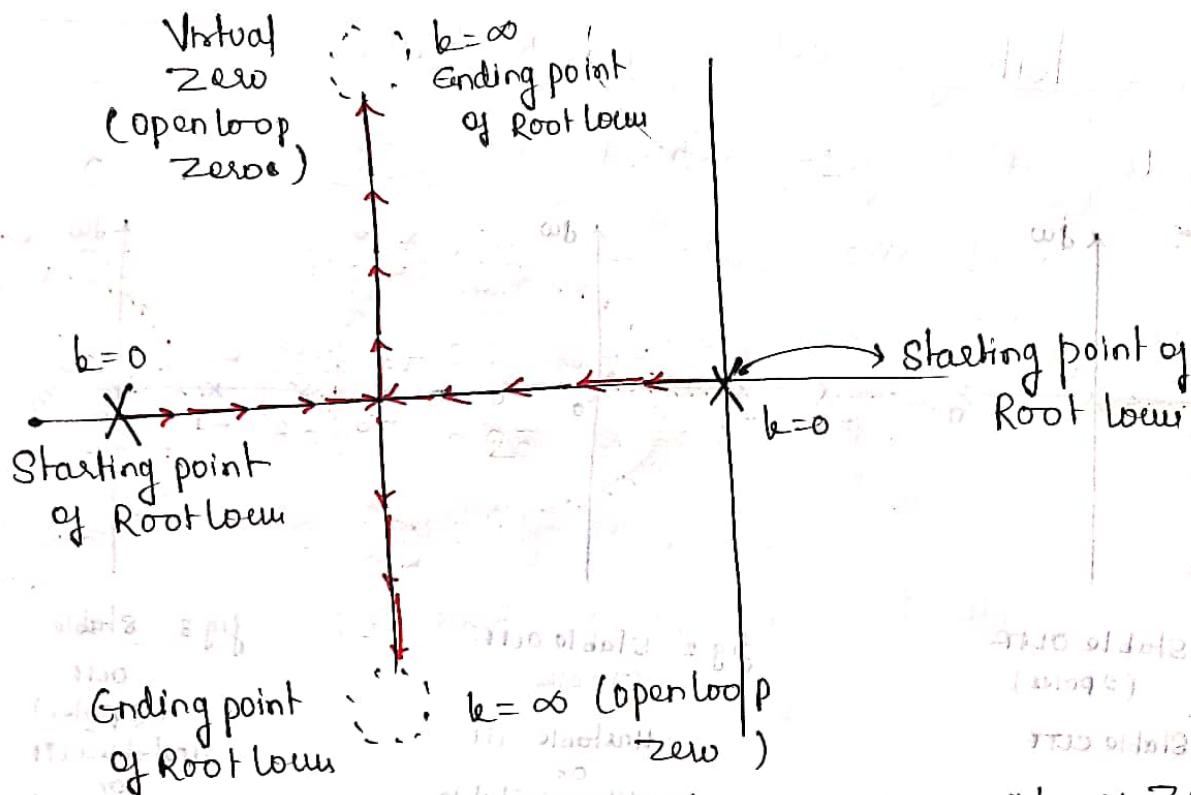
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- In Case of proper  $\frac{P}{Z}$  function ( $\text{No. of poles} > \text{No. of zeros}$ ),

No. of Virtual Zeros is given by

$$Z^* = P - Z$$

$$\text{eg. } G_H = \frac{k}{s^2 + s + 1}$$

$$P = 2, Z = 0$$

$$Z^* = 2 - 0 = 2$$

$$0 = (2H)$$

Hence No. of Virtual Zeros are

$$2$$

$$\frac{(2H)}{(2H+1)}$$

$$0 = (2H)(2H+1) \Rightarrow 2H(2H+1) = 0$$

$$0 = \frac{(2H)^2 + 2H}{2H}$$

$$0 = (2H)^2 + 2H = 0$$

$0 = 2H$  gives half of damping pair

$$0 = (2H)$$

Find poles of this mass-spring system

# Rules for Sketching the Root locus

1. Root locus is always symmetrical about real axis.

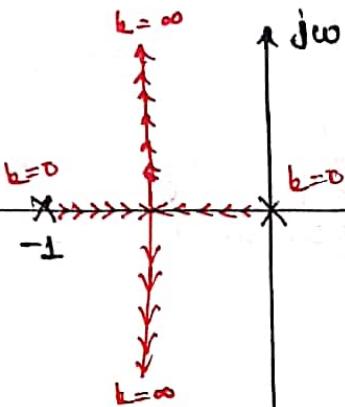


fig 1 Stable OLT  
(2 poles)

Stable CLTF

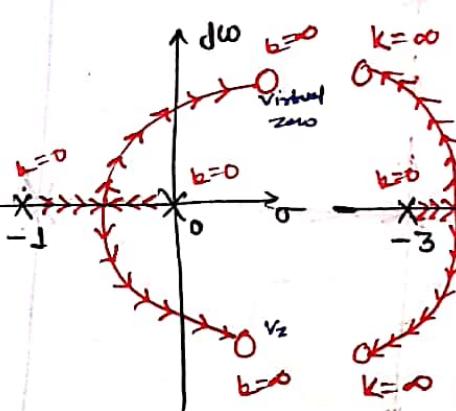


fig 2 Stable OLT  
(3 poles)

Unstable CLTF

or  
Conditional Stable

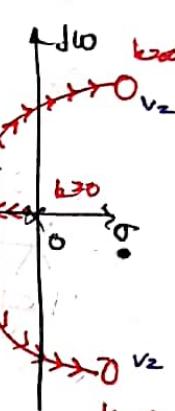


fig 3 Stable  
OLT  
(4 poles)

Unstable CLTF  
or  
Conditional stable

2. Root locus always starts from open loop pole ( $k=0$ ) and terminate either finite open loop zero or infinity i.e. Virtual zero ( $k=\infty$ ).

Key point :- If  $P > Z$  the Root locus will terminate at  $\infty$  (Virtual zero)

Consider

$$G(s)H(s) = \frac{k N(s)}{D(s)}$$

Zeros of OLT  $N(s) = 0$

Poles of OLT  $D(s) = 0$

$$\text{CLTF} = \frac{G(s)}{1 + G(s)H(s)}$$

$$\text{Poles of CLTF} = 1 + G(s)H(s) = 0$$

$$\Rightarrow 1 + k \frac{N(s)}{D(s)} = 0$$

$$\therefore \frac{D(s) + kN(s)}{D(s)} = 0 \quad \text{--- (2)}$$

Starting point of Root locus  $\frac{D(s)}{D(s)} = 0$

$$D(s) = 0$$

Hence Root locus starts with Poles of OLT.

## Chapter - 6 Polar plot

Plot	Variable	Range
Root locus	$k$	$0 < k < \infty$
Inverse Root locus	$k$	$-\infty < k < 0$
Root Contour	$k$	$-\infty < k < \infty$
Polar plot	$\omega$	$0 < \omega < \infty$
Nyquist plot	$\omega$	$-\infty < \omega < \infty$

Ques. i)  $G_H(s) = \frac{1}{s}$

ii)  $G_H(s) = \frac{1}{s^2}$

iii)  $G_H(s) = \frac{1}{s^3}$

iv)  $G_H(s) = \frac{1}{s^4}$

i)  $G_H(s) = 1/s$

put  $s = j\omega$

$G_H(j\omega) = 1/j\omega$

$|G_H(j\omega)| = \frac{1}{\omega}$  (Magnitude)

$\angle G_H(j\omega) = -90^\circ$  (phase)

$\angle j = 90^\circ$

$\angle \frac{1}{j} = -90^\circ$

$\angle j^2 = 180^\circ$

$\angle 1/j_2 = -180^\circ$

$\angle j^3 = 270^\circ$

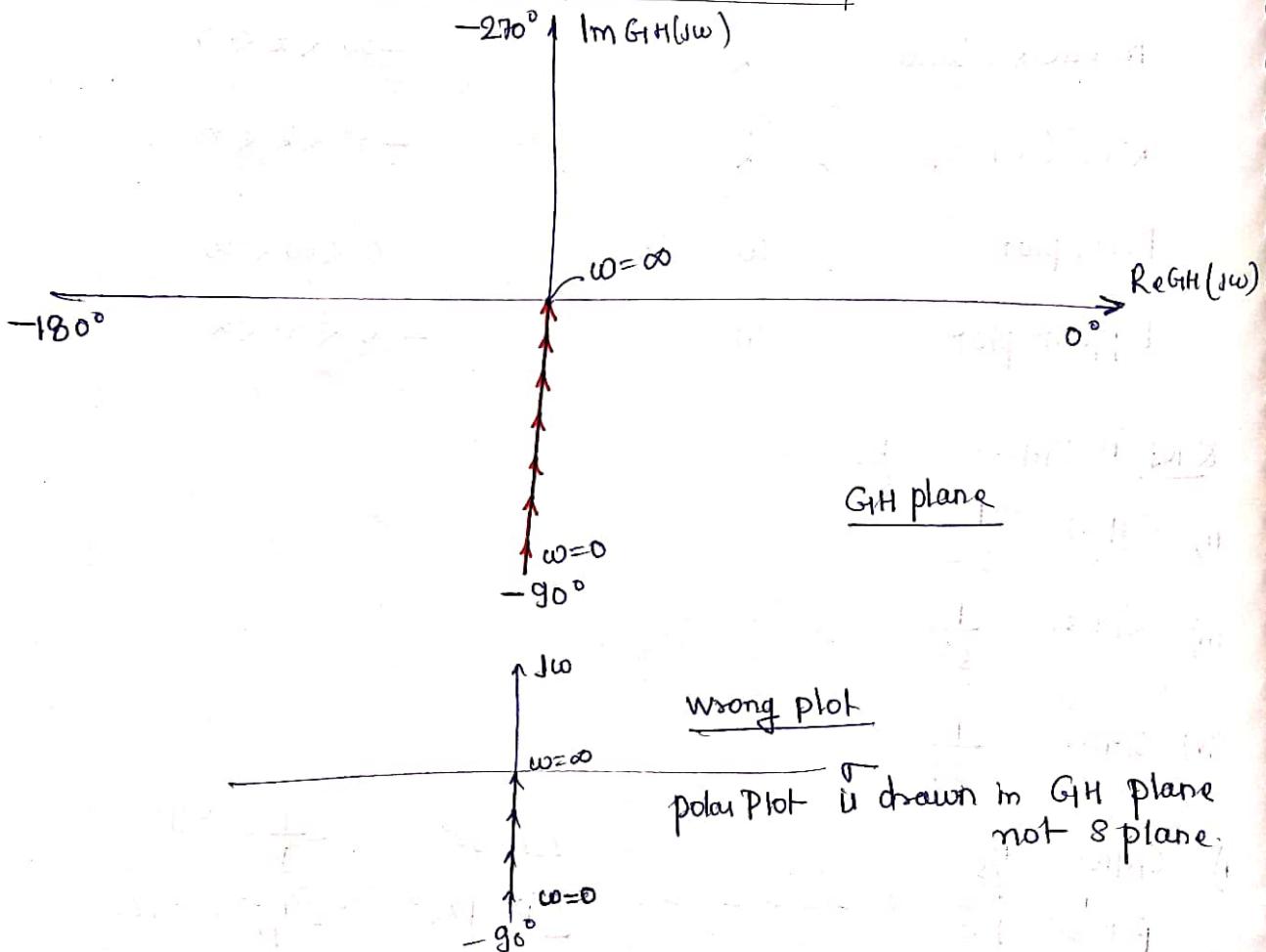
$\angle 1/j_3 = -270^\circ$

$\angle j^4 = 360^\circ$

$\angle 1/j_4 = -360^\circ$

$\omega$	$ G_{IH}(j\omega) $	$\angle G_{IH}(j\omega)$
0	$\infty$	$-90^\circ$
$\infty$	0	$-90^\circ$

$$\text{for } G_{IH}(s) = \frac{1}{s}$$



wrong plot

polar plot is drawn in G<sub>IH</sub> plane  
not s plane

$$ii) G_{IH} = \frac{1}{s^2}$$

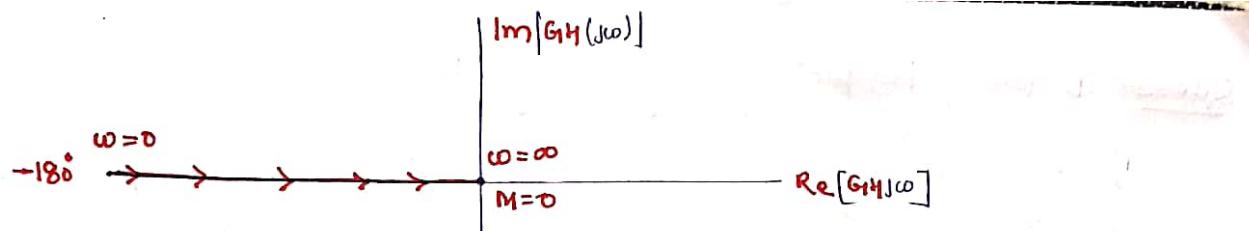
$$\text{Sol } s \rightarrow j\omega$$

$$G_{IH}(j\omega) = \frac{1}{(j\omega)^2}$$

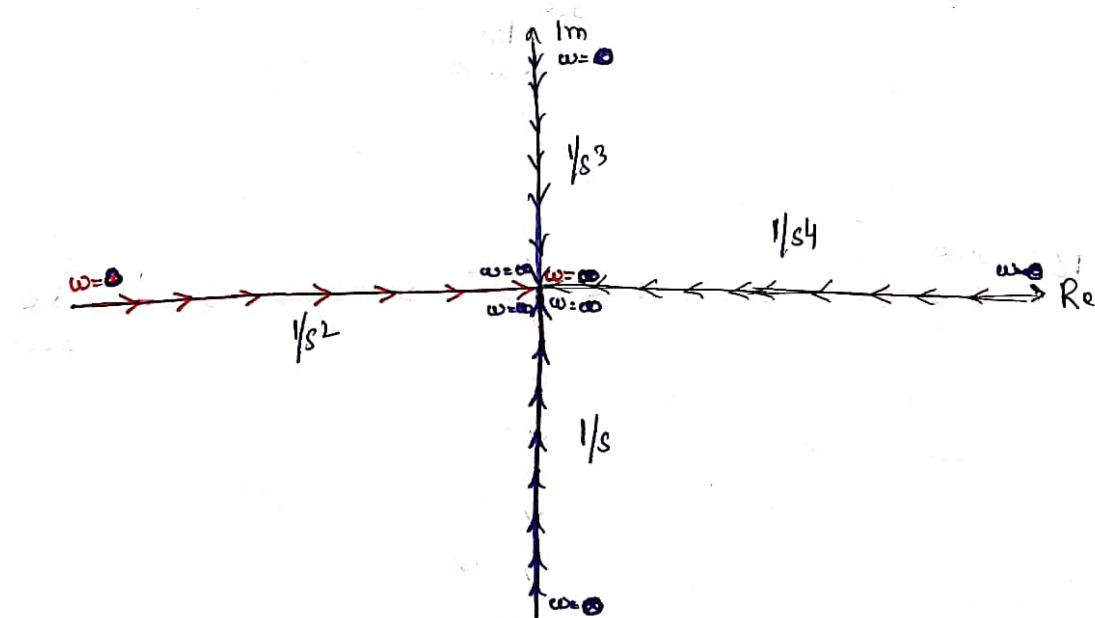
$$|G_{IH}(j\omega)| = \frac{1}{\omega^2}$$

$$\angle G_{IH}(j\omega) = -180^\circ$$

$\omega$	$ M $	$\phi$
0	$\infty$	$-180^\circ$
$\infty$	0	$-180^\circ$



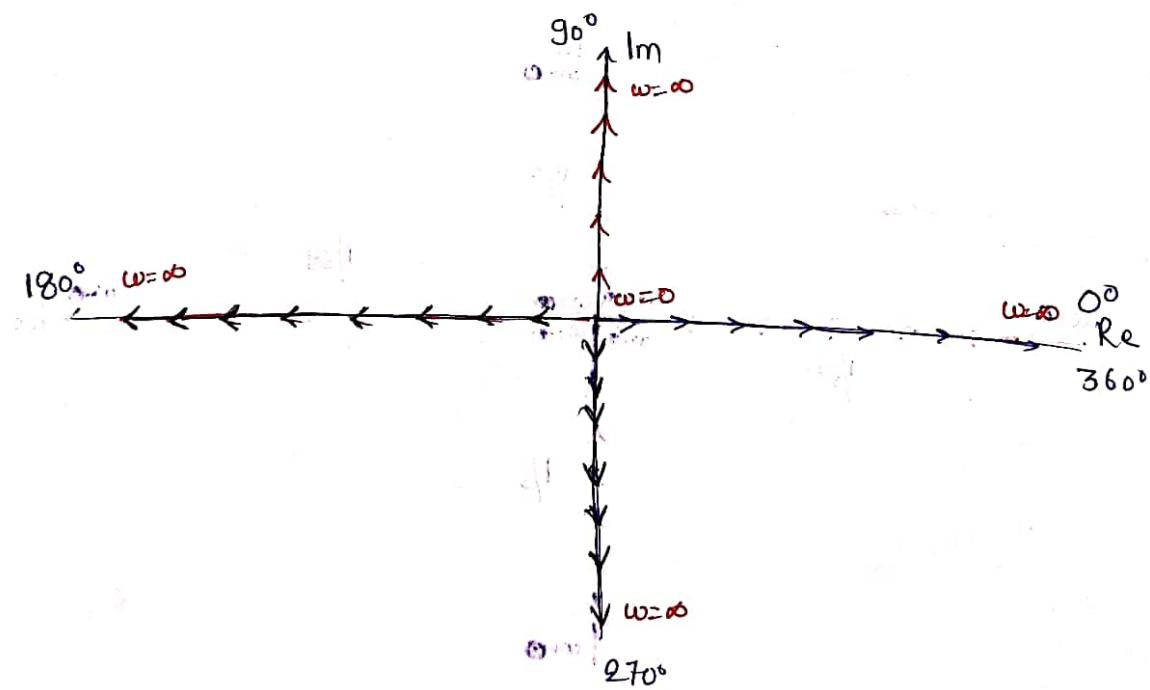
$G(s)H(s)$	$G_H(j\omega)$	$ G_H(j\omega) $	$\angle G_H(j\omega)$	$ G_H(j0) $	$ G_H(j\infty) $
$\frac{1}{s}$	$\frac{1}{j\omega}$	$\frac{1}{\omega}$	$-90^\circ$	$\infty$	0
$\frac{1}{s^2}$	$\frac{1}{(j\omega)^2}$	$\frac{1}{\omega^2}$	$-180^\circ$	$\infty$	0
$\frac{1}{s^3}$	$\frac{1}{(j\omega)^3}$	$\frac{1}{\omega^3}$	$-270^\circ$	$\infty$	0
$\frac{1}{s^4}$	$\frac{1}{(j\omega)^4}$	$\frac{1}{\omega^4}$	$-360^\circ$	$\infty$	0



Ques Draw polar plot for following OLT

- i)  $S$
- ii)  $S^2$
- iii)  $S^3$
- iv)  $S^4$

$G_H(s)$	$G_H(j\omega)$	$ G_H(j\omega) $	$\angle G_H(j\omega)$	$ G_H(j0) $	$ G_H(j\infty) $
$S$	$j\omega$	$\infty$	$90^\circ$	0	$\infty$
$S^2$	$(j\omega)^2$	$\omega^2$	$180^\circ$	0	$\infty$
$S^3$	$(j\omega)^3$	$\omega^3$	$270^\circ$	0	$\infty$
$S^4$	$(j\omega)^4$	$\omega^4$	$360^\circ$	0	$\infty$



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1. Previous year paper 4-5 times practice before final exam.
2. Subject wise study reference STD book
3. Test series practice more n more (Try to latest test series 2-3 fully solve then join online test series.)

**Noted-: Single Source Follow, Revise**

**Multiple Time Best key of Success**

## \* Polar Plot for Type zero System

Type zero system represents no poles in origin in OLT.

$G_H(s)$	Type	Order
$\frac{1}{s+1}$	0	1
$\frac{1}{(s+1)(s+2)}$	0	2
$\frac{1}{(s+1)(s+2)(s+3)}$	0	3
$\frac{1}{(s+1)(s+2)(s+3)(s+4)}$	0	4

Solt  $G_H(s) = \frac{1}{s+1}$

put  $s = j\omega$

$$G_H(j\omega) = \frac{1}{j\omega + 1}$$

$$|G_H(j\omega)| = \frac{1}{\sqrt{\omega^2 + 1}}$$

$$\angle G_H(j\omega) = -\tan^{-1}\omega$$

At  $\omega = 0$

$$|G_H(j\omega)| = 1$$

$$\angle G_H(j\omega) = 0^\circ$$

At  $\omega = 1$

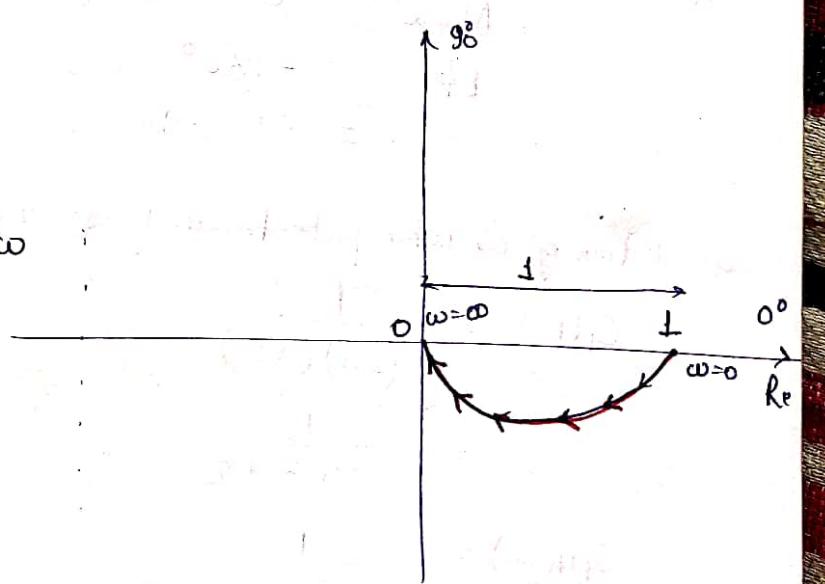
$$|G_H(j\omega)| = 0.707$$

$$\angle G_H(j\omega) = -45^\circ$$

At  $\omega = \infty$

$$|G_H(j\omega)| = 0$$

$$\angle G_H(j\omega) = -90^\circ$$



$\omega$	$ G_H(j\omega) $	$\angle G_H(j\omega)$
0	1	$0^\circ$
1	$1/\sqrt{2}$	$-45^\circ$
$\infty$	0	$-90^\circ$

$$Q12 \quad G_H(s) = \frac{1}{(s+1)(s+2)}$$

$$G_H(j\omega) = \frac{1}{(j\omega+1)(j\omega+2)}$$

$$|M| = \frac{1}{\sqrt{\omega^2+1} \sqrt{\omega^2+4}}$$

$$= \frac{1}{\sqrt{(\omega^2+1)(\omega^2+4)}}$$

$$\angle \phi = -\tan^{-1} \frac{\omega}{1} - \tan^{-1} \frac{\omega}{2}$$

At  $\omega = 0$

$$M_{\omega=0} = \frac{1}{2}$$

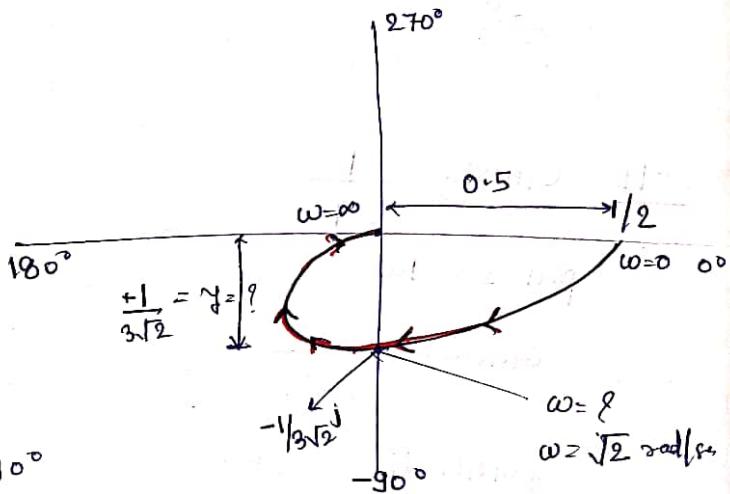
$$\angle \phi_{\omega=0} = 0^\circ$$

At  $\omega = \infty$

$$M_{\omega=\infty} = 0$$

$$\begin{aligned} \angle \phi_{\infty} &= -180^\circ \\ &= -90^\circ - 90^\circ \end{aligned}$$

$\omega$	$ G_H $	$\angle G_H$
0	$1/2$	$0^\circ$
$\infty$	0	$-90^\circ - 90^\circ = -180^\circ$



Calculation of  $\omega$  when plot intersect  $-90^\circ$  axis

$$\begin{aligned} G_H(s) &= \frac{1}{(s+1)(s+2)} \\ &= \frac{1}{s^2 + 3s + 2} \end{aligned}$$

$$G_H(j\omega) = \frac{1}{-\omega^2 - 3j\omega + 2}$$

Numerator Const. Hence Real part  
 $\operatorname{Re}[G_H(j\omega)] = 0$

$$\omega^2 - \omega^2 = 0 \quad \text{gives Imaginary value}$$

$$\omega = \pm \sqrt{2}$$

always +ve

$$\begin{aligned} G_H(j\omega) &= \frac{1}{2j\omega} \\ &= \frac{1}{2\sqrt{2}j} = -\frac{1}{2\sqrt{2}}j \end{aligned}$$

Hence distance from orig.

$$y = -1/\sqrt{2}$$

$$\text{Sol3} \quad G_H = \frac{10}{(s+1)(s+2)(s+3)}$$

$$G_H(j\omega) = \frac{10}{(j\omega+1)(j\omega+2)(j\omega+3)}$$

$$M = \frac{10}{\sqrt{(\omega^2+1)(\omega^2+4)(\omega^2+9)}}$$

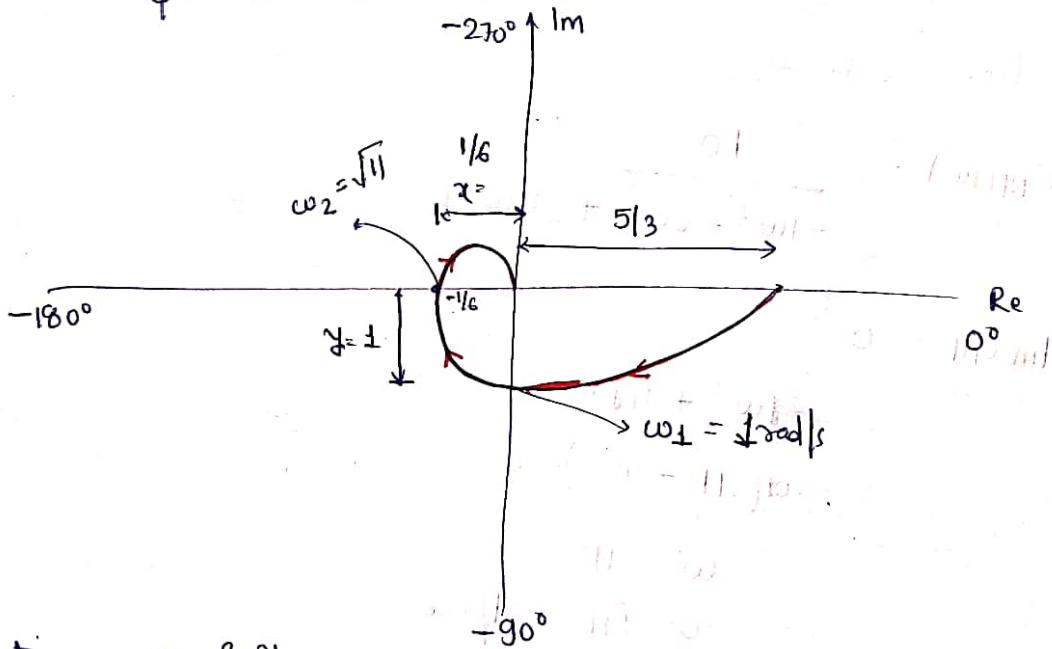
$$\phi = 0^\circ - \tan^{-1}\omega - \tan^{-1}2 - \tan^{-1}3$$

$$\text{at } \omega=0 \quad M = \frac{10}{6} = \frac{5}{3}$$

$$\phi = 0^\circ$$

$$\text{at } \omega=\infty \quad M = 0$$

$$\phi = -90^\circ - 90^\circ - 90^\circ = -270^\circ$$



Calculation:  $\omega_1$  &  $\gamma$

$$G_H(s) = \frac{10}{(s+1)(s^2+5s+6)}$$

$$= \frac{10}{s^3 + 6s^2 + 11s + 6}$$

$\omega$	$ G_H $	$\angle G_H$
0	$5/3$	$0^\circ$
$\infty$	0	$-90^\circ - 90^\circ - 90^\circ = -270^\circ$

$$G_H(j\omega) = \frac{10}{-j\omega^3 - 6\omega^2 + j11\omega + 6}$$

$$\operatorname{Re}[G_H(j\omega)] = 0$$

$$6\omega^2 - 6 = 0$$

$$\omega^2 = 6/6$$

$$\omega = 1 \text{ rad/sec}$$

$$G_H(j\omega) = \frac{10}{-j - 6 + j11 + 6}$$

$$= \frac{10}{10j}$$

$$= -j1$$

Calculation  $\propto \ln \omega_2$

$$G_H(j\omega) = \frac{10}{-j\omega^3 - 6\omega^2 + j11\omega + 6}$$

$$\operatorname{Im} G_H = 0$$

$$= -j\omega^3 + j11\omega = 0$$

$$\Rightarrow \cos(11 - \omega^2) = 0$$

$$\omega^2 = 11$$

$$\omega = \sqrt{11} \text{ rad/sec}$$

$$\operatorname{Real} G_H \text{ at } \omega = \sqrt{11}$$

$$= \frac{10}{-6 \times 11 + 6}$$

$$= \frac{10}{-66 + 6} = \frac{10}{-60} = -1/6$$

$$x = -1/6$$

# Nyquist Plot

- In Nyquist plot variable is entire S-plane while in Polar plot the variable is positive frequency line of S plane.
- In case of Nyquist plot both +ve as well as -ve frequency exist while in case of polar plot only +ve frequency exist.
- Nyquist plot describes both absolute & relative stability while polar plot describes only relative stability.
- Nyquist plot is used to draw complete frequency response of OLT.
- \* Nyquist stability criteria describe closed system stability from open loop frequency response and open loop poles.
- Nyquist plot is used to find number of closed loop poles in the right hand side of S-plane which is not possible in polar plot.
- Nyquist plot is used to find range of 'k' for system stability.
- Nyquist plot is used to find relative stability parameter ( $\omega_{pc}$ ,  $\omega_{ge}$ ,  $G_M$ ,  $PM$ )
- Consider OLT

$$G(s) = \frac{k N(s)}{\Omega(s)} \quad \text{--- (1)}$$

$$G_E = 1 + G_H = 1 + \frac{k N(s)}{\Omega(s)} = \frac{\Omega(s) + N(s)k}{\Omega(s)} \quad \text{--- (2)}$$

$$C_D F = \frac{G_H}{1 + G_H} = \frac{k N(s) / \Omega(s)}{1 + k N(s) / \Omega(s)} = \frac{k N(s)}{\Omega(s) + k N(s)} \quad \text{--- (3)}$$

from (1) & (3)

Poles of  $C_{eq}^n$  (2) & (3)

F

Poles of  $C_{eq}^n$  = Poles of OLTF

→ (4)

from (1) & (3)

Zeroes of chann. eq<sup>n</sup> = poles of CLTF

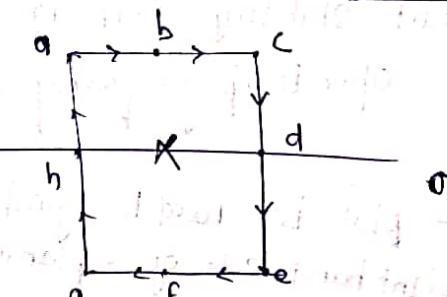
→ (5)

Ex 01

$$G_1(s) = \frac{1}{s-2}$$

jw

S plane



Point	$G_H = \frac{1}{s-2}$	$ G_1(jw) $	$\angle G_1(s)$
a = 1 + j1	$\frac{1}{j1-2}$	$1/\sqrt{2}$	-135°
b = 2 + j1	$\frac{1}{j1}$	1	-90°
c = 3 + j1	$\frac{1}{1+j1}$	$1/\sqrt{2}$	-45°
d = 3 + j0	$\frac{1}{1}$	1	0°
e = 3 - j1	$1/j1-1$	$1/\sqrt{2}$	45°
f = 2 - j1	$1/-j1$	1	90°
g = 1 - j1	$1/-1-j1$	$1/\sqrt{2}$	-225° (135°)
h = 1 - j0	$1/-1$	1	±180°

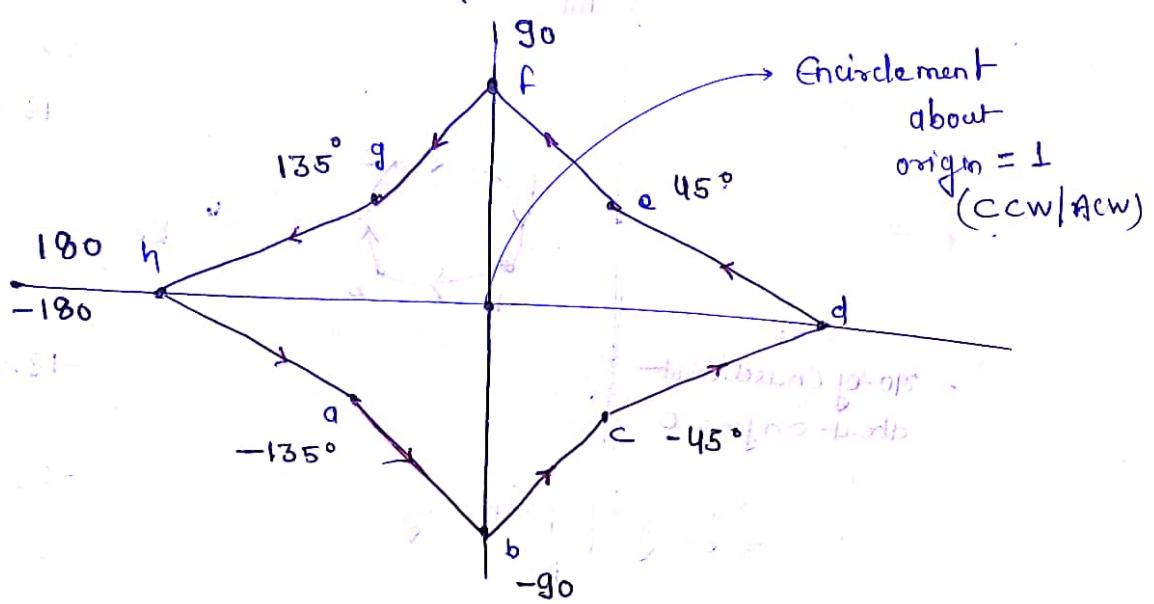
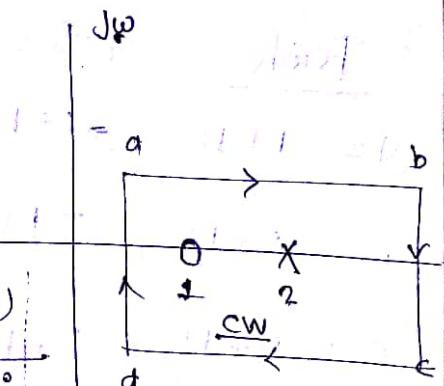


fig. freq plane of  $G_1(s)$   
polar plot of  $G_1(s)$

$$C.E = |1 + G_1(s)| = 1 + \frac{1}{s-2} = \frac{s-2+1}{s-2}$$

$$1 + G_1(s) = \frac{s-1}{s-2}$$

Points	$1 + G_1(s) = \frac{s-1}{s-2}$	$ 1 + G_1(s) $	$\angle(1 + G_1(s))$
$a = 0.5 + j1$	$\frac{-0.5 + j1}{-1.5 + j1}$	0.62	-29.74°
$b = 3 + j1$	$\frac{2 + j1}{1 + j1}$	1.58	-18.43°
$c = 3 - j1$	$\frac{2 - j1}{1 - j1}$	1.58	18.45°
$d = 0.5 - j1$	$\frac{-0.5 - j1}{-1.5 - j1}$	0.62	29.74°



S-plane of  $1 + G_1(s)$

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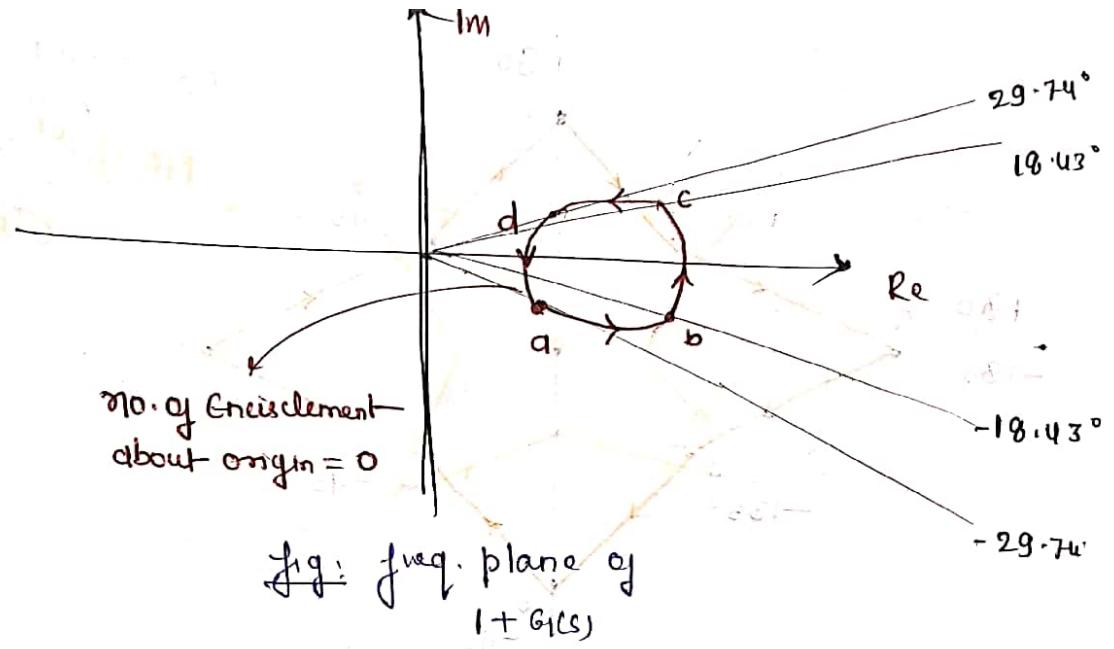
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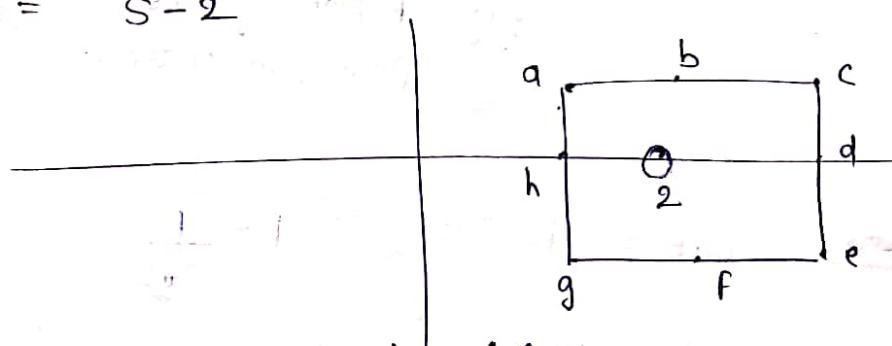
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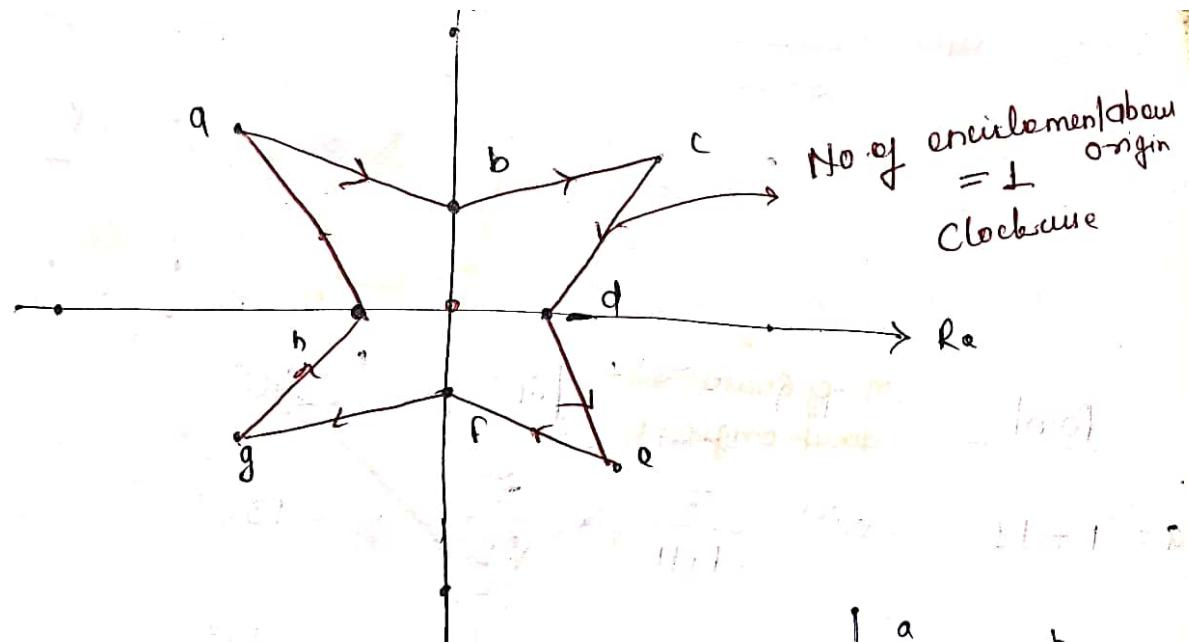
**Multiple Time Best key of Success**



$$\underline{QX_2} \quad G_1(s) = \quad s - 2$$



Points	$G(s) = s - 2$	$ G(s) $	$\angle G(s)$
a = $1 + j1$	$-1 + j1$	$\sqrt{2}$	$135^\circ$
b = $2 + j1$	$+j1$	1	$90^\circ$
c = $3 + j1$	$1 + j1$	$\sqrt{2}$	$45^\circ$
d = $3 + j0$	$1 + j0$	1	$0^\circ$
e = $3 - j1$	$1 - j1$	$\sqrt{2}$	<del><math>45^\circ</math></del> / $-45^\circ$
f = $2 - j1$	$0 - j1$	1	$-90^\circ$
g = $1 - j1$	$-1 - j1$	$\sqrt{2}$	$+225^\circ$
h = $1 - j0$	$-1 - j0$	1	$-180^\circ$



$$C.E. = 1 + G_1(s)$$

$$\Rightarrow 1 + s - 2$$

$$= s - 1$$

Points

$$1 + G_1(s) \doteq s - 1$$

$$|1 + G_1(s)|$$

$$\angle 1 + G_1(s)$$

$$a = 0.5 + j1$$

$$-0.5 + j1$$

$$\sqrt{2}$$

$$116.57^\circ$$

$$b = 2 + j1$$

$$1 + j1$$

$$\sqrt{2}$$

$$45^\circ$$

$$c = 2 - j1$$

$$1 - j1$$

$$\sqrt{2}$$

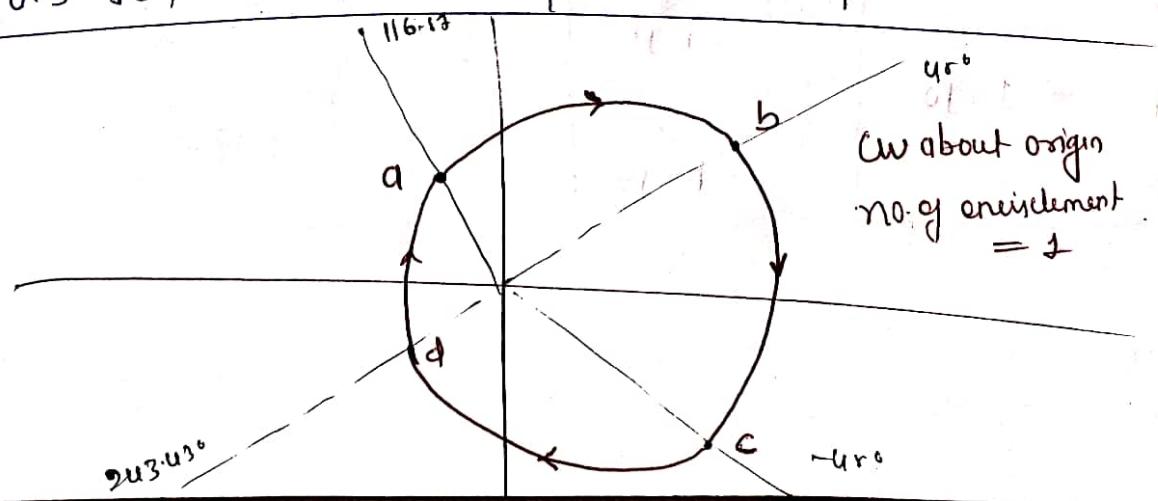
$$-45^\circ$$

$$d = 0.5 - j1$$

$$-0.5 - j1$$

$$\sqrt{2}$$

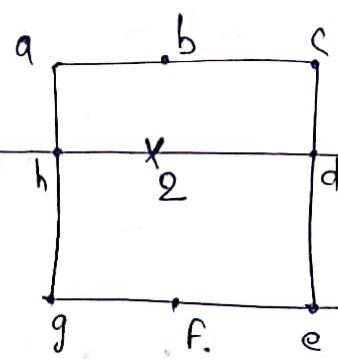
$$243.43^\circ$$



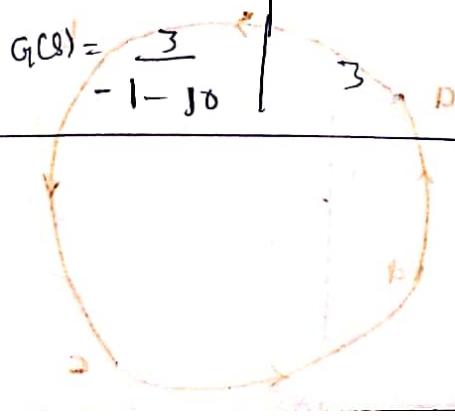
Ans

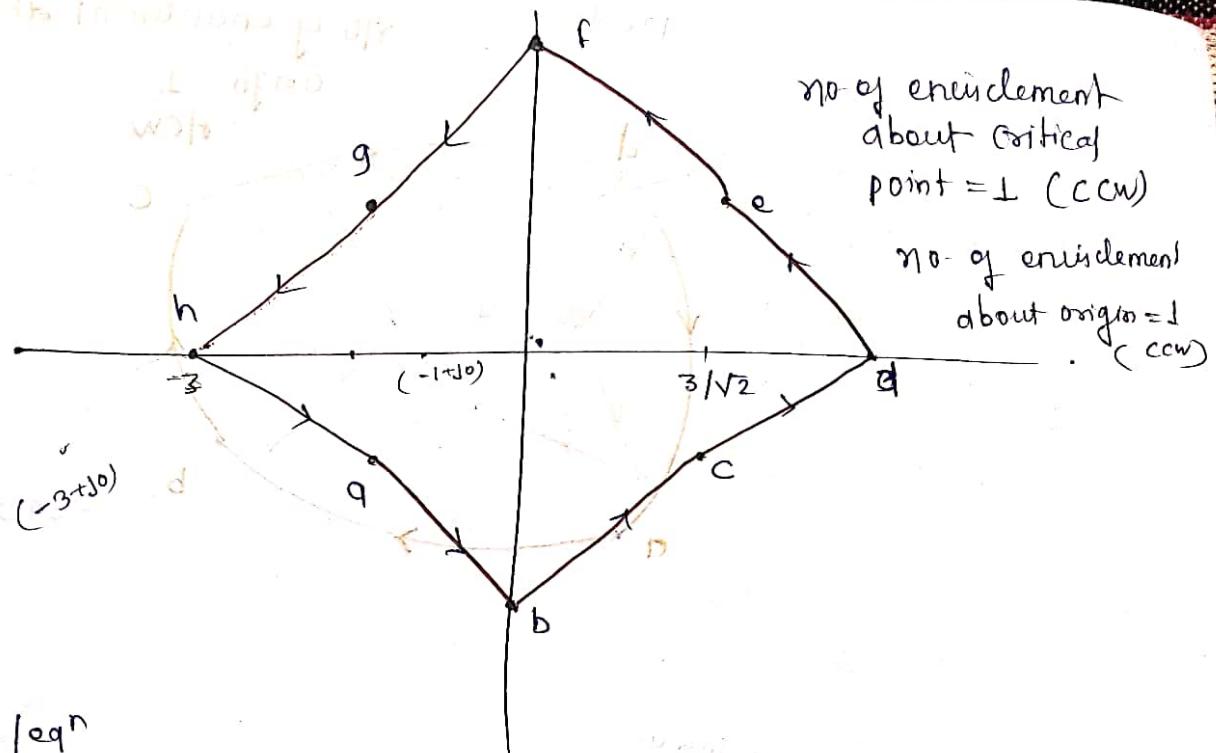
$$G_1(s) = \frac{3}{s-2}$$

*(points)*  
 $L = 6^{\circ}$   
 $\tan(\theta)$



<u>Points</u>	$G_1(s) = \frac{3}{s-2}$	$ G_1(s) $	$\angle G_1(s)$
$a = 1+j1$	$G_1(s) = \frac{3}{-1+j1}$	$\frac{3}{\sqrt{2}}$	$-135^{\circ}$
$b = 2+j1$	$G_1(s) = \frac{3}{j1}$	$3$	$-90^{\circ}$
$c = 3+j1$	$G_1(s) = \frac{3}{1+j1}$	$\frac{3}{\sqrt{2}}$	$-45^{\circ}$
$d = 3+j0$	$G_1(s) = \frac{3}{1+j0}$	$\frac{3}{\sqrt{2}} = 3$	$0^{\circ}$
$e = 3-j1$	$G_1(s) = \frac{3}{1-j1}$	$\frac{3}{\sqrt{2}}$	$45^{\circ}$
$f = 2-j1$	$G_1(s) = \frac{3}{0-j1}$	$3$	$90^{\circ}$
$g = 1-j1$	$G_1(s) = \frac{3}{-1-j1}$	$\frac{3}{\sqrt{2}}$	$-225^{\circ} (= 135^{\circ})$
$h = 1+j0$	$G_1(s) = \frac{3}{-1+j0}$	$3$	$\pm 180^{\circ}$





C. /eq^n

$$1 + G_1(s)$$

$$= \frac{(s+1) + \frac{3}{s-2}}{s-2} \quad (2) \text{ if } s \neq 2 \quad (\text{because } s-2 \neq 0)$$

↓  
cancel

$$= \frac{s-2+3}{s-2} \quad (\text{cancel } s-2 \text{ from numerator and denominator})$$

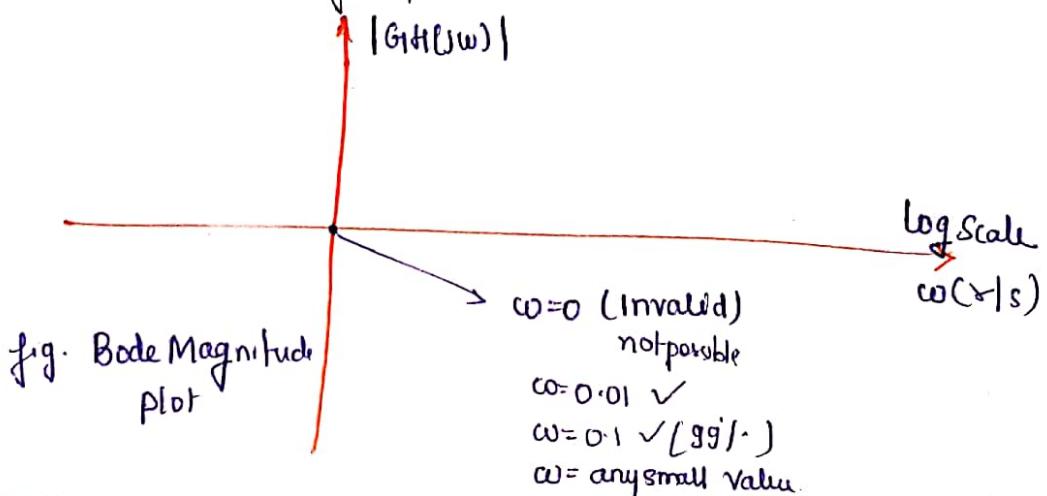
↓  
cancel

$$= \frac{s+1}{s-2} \quad (\text{cancel } 3 \text{ from numerator and denominator})$$

points	$1+G_1(s) = \frac{8+j1}{s-2}$	$ 1+G_1(s) $	$\angle(1+G_1(s))$
$a = 1+j1$	$\frac{2+j1}{-1+j1}$	$\sqrt{5}$	-108.43
$b = 3+j1$	$\frac{4+j1}{j+1}$	2.91	-30.96
$c = 3-j1$	$\frac{4-j1}{1-j1}$	2.91	30.96
$d = 1-j1$	$\frac{2-j1}{-1-j1}$	$\sqrt{5}$	108.43

## BODE PLOT

- Used to determine frequency response of OLTF
- Determines the stability of CLTF by freq. response of OLTF
- Used to find relative stability parameter ( $\omega_{pc}$ ,  $\omega_{gc}$ ,  $G_M$  PM)
- Addition of poles and zeros can easily be analysed with the help of Bodeplot.
- \* • Bode plot is analysed only for MPS (stable OLTF)
- Bode plot is combination of two individual plot.  
Magnitude & phase plot.
- It is plotted on Semi Log paper.
- In Bode plot, Bode Magnitude plot is a logarithmic plot.  
(Magnitude in dB)
- In Bode Magnitude plot  $\log 0$  is invalid quantity So it is impossible to calculate magnitude at  $\omega=0$ .
- In Bode phase plot, Phase at  $\omega=0$  is valid quantity  
So it is possible to calculate phase angle at  $\omega=0$ .  
but Magnitude and phase plot is drawn for common axis. So  $\omega=0$  is invalid for Magnitude plot  
it is also invalid for phase plot.



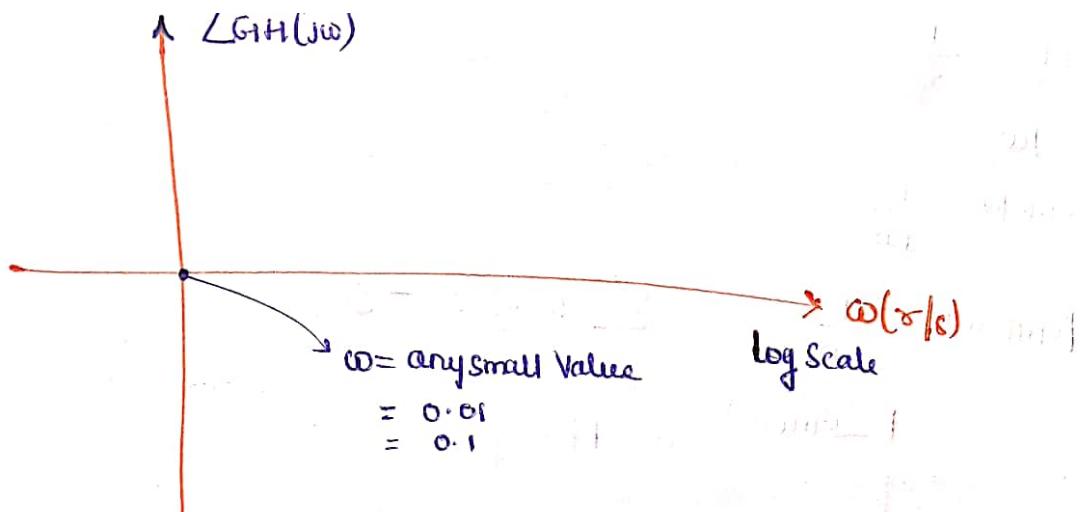


fig. Bode phase plot

### The Concept of Minimum & Non Minimum phase System

1. In Case of MPS all poles and zeros should be located in the LHS of S-plane.
2. MPS is always stable system.
3. In Case of NMPS, it contain at least one pole or zero in the right hand Side of Splane.
4. NMPS may be unstable( in case of RH pole).
5. Mostly RH zero System is referred as NMPS.

Ques Draw Bodeplot for following OLTF

i)  $G_H(s) = \frac{1}{s}$

ii)  $G_H(s) = \frac{1}{s^2}$

iii)  $G_H(s) = \frac{1}{s^3}$

iv)  $G_H(s) = \frac{1}{s^4}$

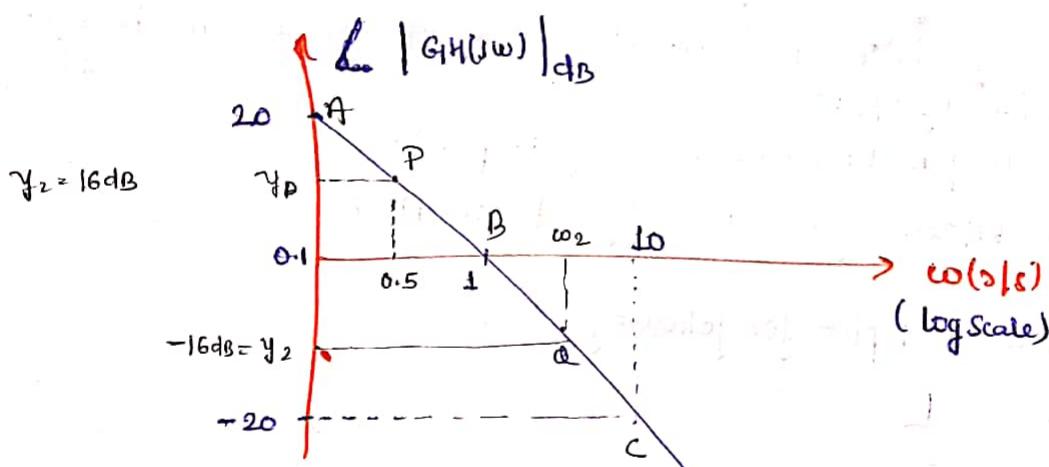
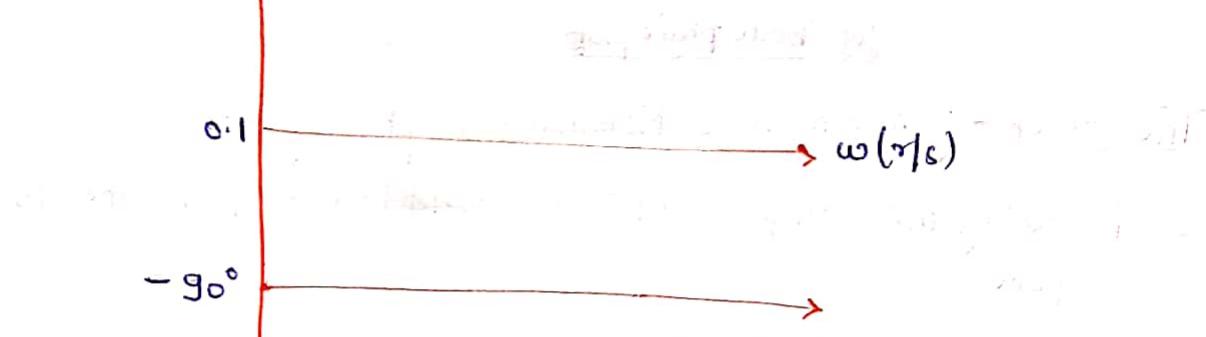
$$\text{Sol L } G_H(s) = \frac{1}{s}$$

$$s = j\omega$$

$$G_H(j\omega) = \frac{1}{j\omega}$$

$$|G_H(j\omega)| = \frac{1}{\omega} \quad \angle G_H(j\omega) = -90^\circ$$

$\angle G_H(j\omega)$  Bode Phase plot



$$|G_H(j\omega)| = \frac{1}{\omega}$$

$$= -20 \log \omega$$

$\omega$	$ G_H(j\omega) _{\text{dB}}$
0.1	$-20 \log 0.1 = 20 \text{ dB}$
1.0	$-20 \log 1 = 0 \text{ dB}$
10	$-20 \log 10 = -20 \text{ dB}$

$$\left. \begin{aligned} \log 1 &= 0 \\ \log 10 &= 1 \\ \log 10^2 &= 2 \\ \log 1000 &= 3 \\ \log 0.1 &= -1 \\ \log 0.01 &= -2 \\ \log 0.001 &= -3 \end{aligned} \right\}$$

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**Noted-:** Dear Aspirants If you do practice previous year paper 50% your work finished.so Guys daily at least 30minutes give previous year.

1. Previous year paper 4-5 times practice before final exam.
2. Subject wise study reference STD book
3. Test series practice more n more (Try to latest test series 2-3 fully solve then join online test series.)

**Noted-: Single Source Follow, Revise**

**Multiple Time Best key of Success**

$$\left. \begin{array}{l} \omega_2 = 10\omega_1 \\ \omega_3 = 10\omega_2 \end{array} \right] \text{Decade frequency}$$

$$\left. \begin{array}{l} \omega_2 = 2\omega_1 \\ \omega_3 = 2\omega_2 \end{array} \right] \text{Octave freq.}$$

$$\omega_2 = 10\omega_1$$

$$\frac{\omega_2}{\omega_1} = 10$$

$$20 \log \frac{\omega_2}{\omega_1} = 20 \log 10 = 20 \text{dB/dec}$$

$$\frac{\omega_2}{\omega_1} = 2$$

$$20 \log \frac{\omega_2}{\omega_1} = 20 \log 2 = 6 \text{dB/oct}$$

$$6 \text{dB/oct} = 20 \text{dB/dec}$$

$$12 \text{dB/oct} = 40 \text{dB/dec}$$

$$18 \text{dB/oct} = 60 \text{dB/dec}$$

$$-24 \text{dB/oct} = -80 \text{dB/dec}$$

$$-30 \text{dB/oct} = -100 \text{dB/dec}$$

$$\text{Slope}_{AB} = \frac{Y_B - Y_A}{\log \omega_B - \log \omega_A} = \frac{0 - 20}{\log \left( \frac{\omega_B}{\omega_A} \right)} = \frac{-20}{\log \left( \frac{1}{0.1} \right)} = -20 \text{dB/dec}$$

$$\text{Slope}_{BC} = \frac{Y_C - Y_B}{\log \omega_C - \log \omega_B} = \frac{-20 - 0}{\log \left( \frac{\omega_C}{\omega_B} \right)} = \frac{-20}{\log \left( \frac{10}{1} \right)} = -20 \text{dB/dec}$$

$$\text{Slope}_{AC} = \frac{Y_C - Y_A}{\log \omega_C - \log \omega_A} = \frac{-20 - 20}{\log \left( \frac{10}{0.1} \right)} = \frac{-40}{\log 100} = -20 \text{dB/dec}$$

Magnitude at point P:

$$\text{Slope}_{AP} = -20 \text{dB/decad}$$

$$\frac{Y_P - Y_A}{\log \left( \frac{\omega_P}{\omega_A} \right)} = -20$$

$$\Rightarrow \frac{Y_P - 20}{\log 0.5/0.1} = -20 \Rightarrow Y_P = -20 \log 5 + 20 = 6.02 \text{dB}$$

$\omega_2 =$

$$\text{Slope } BQ = \frac{Y_Q - Y_B}{\log(\omega_B)}$$

$$-20 = \frac{-16 - 0}{\log\left(\frac{\omega_2}{0.1}\right)}$$

$$\log \omega_2 = \frac{16}{20}$$

$$\log \omega_2 = \frac{16}{20}$$

$$\log \omega_2 = \frac{16}{20}$$

$$\omega_2 = 0.6309$$

$$\underline{\omega_2} = 6.3 \text{ rad/s}$$

$$\text{Slope } A_Q = \frac{Y_Q - Y_A}{\log\left(\frac{\omega_2}{0.1}\right)}$$

$$-20 = \frac{-10 - 20}{\log(\omega_2/0.1)}$$

$$\log\left(\frac{\omega_2}{0.1}\right) = \frac{-30}{-20} = 1.5$$

$$\frac{\omega_2}{0.1} = 10^{1.5} = 63.09$$

$$\omega_2 = 6.309 \text{ rad/s}$$

Q2

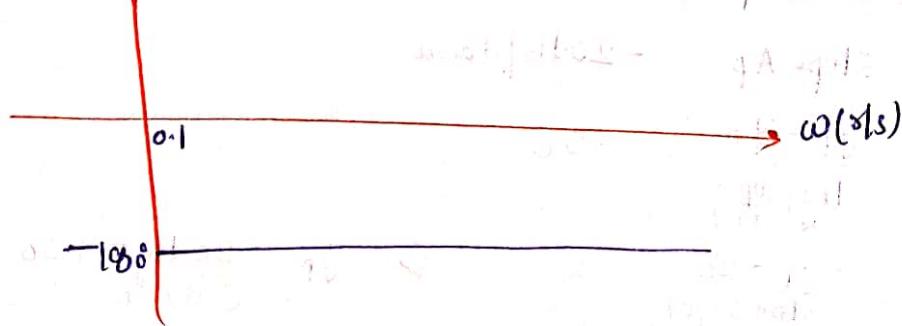
$$G_H(s) = \frac{1}{s^2}$$

$$|G_H(j\omega)| = \frac{1}{\omega^2}$$

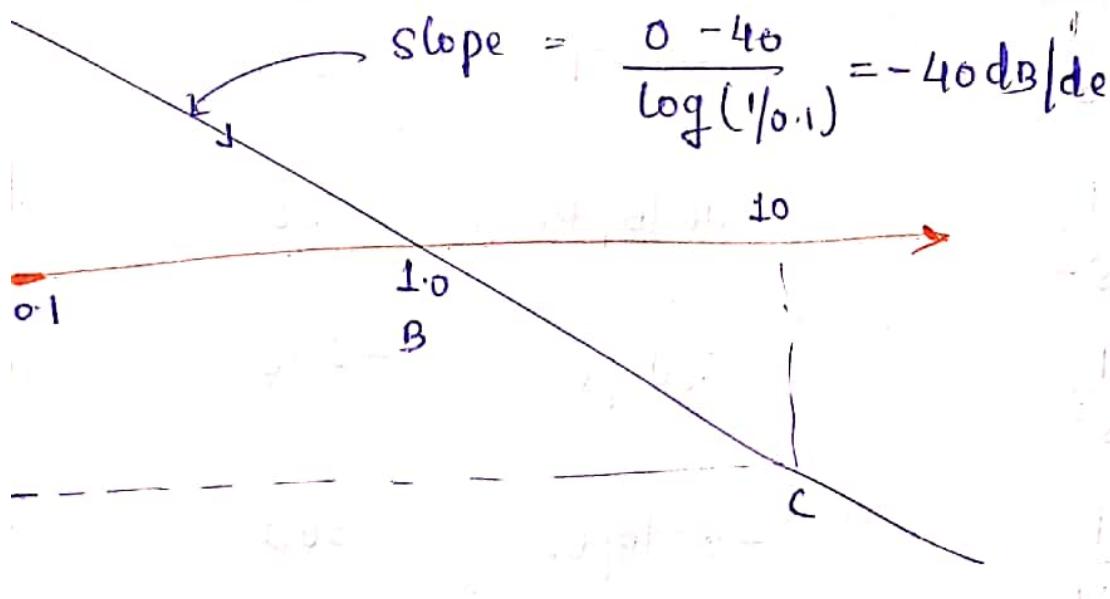
$$\angle G_H(j\omega) = -180^\circ$$

$\omega$	$ G_H(j\omega) $	$\angle G_H(j\omega)$
0.1	100	-180°
1	1	-180°
10	0.01	-180°

$$\angle G_H(j\omega)$$



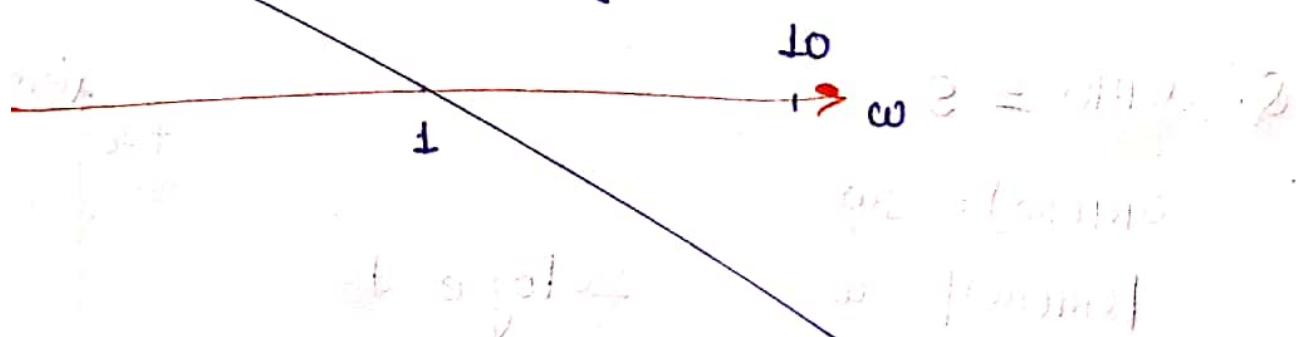
$|G(j\omega)|$



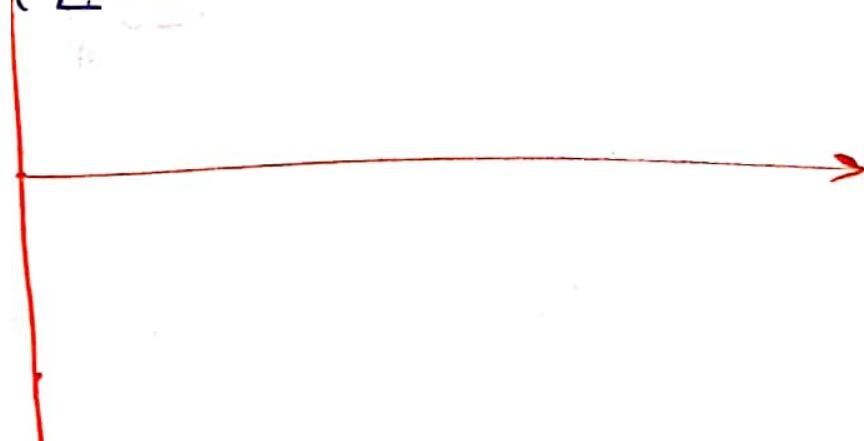
$$\frac{1}{S^3}$$

$|G(j\omega)|$

$$\text{slope} = \frac{0 - 60}{\log(\frac{1}{0.1})} = -60 \text{ dB/dec}$$



$\angle G(j\omega)$



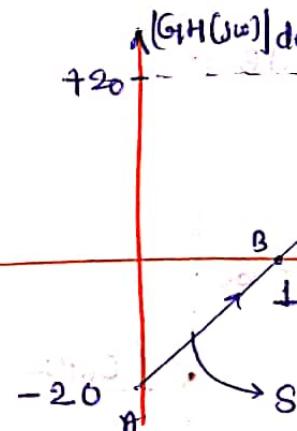
$GH(s)$	Slope	Angle
$\frac{1}{s}$	-20 dB/dec	-90
$\frac{1}{s^2}$	-40 dB/dec	-180
$\frac{1}{s^3}$	-60 dB/dec	-270
$\frac{1}{s^4}$	-80 dB/dec	-360
$\frac{1}{s^n}$	-20n dB/dec	-90n

Q.  $GH(s) = s$

$$GH(j\omega) = j\omega$$

$$|GH(j\omega)| = \omega = 20 \log \omega \text{ dB}$$

$$\angle GH(j\omega) = 90^\circ$$



$G_H(s)$	Slope	Angle
$s$	20 dB/dec	90
$s^2$	40 dB/dec	180
$s^3$	60 dB/dec	270
$s^4$	80 dB/dec	360
$s^n$	$20n$ dB/dec	90n

Que.  $G_H(s) = \frac{k}{s}$

So  $\therefore$  Case 1  $k = 1.0$

$$G_H(s) = \frac{1}{s}$$

Case 1:-

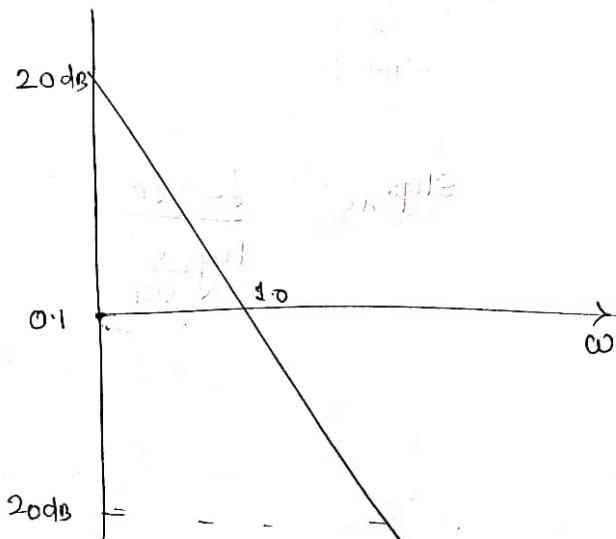
$$k = 0.1 \quad (k < 1)$$

$$G_H(s) = \frac{0.1}{s}$$

$$G_H(j\omega) = \frac{0.1}{j\omega}$$

$$|G_H(j\omega)| = \frac{0.1}{\omega}$$

$$\angle G_H(j\omega) = -90^\circ$$



$$|G_H(j\omega)|_{dB} = 20 \log \left( \frac{0.1}{\omega} \right)$$

$$= 20 \log 0.1 - 20 \log \omega$$

## Unit-9 State Space Analysis

• To study transient & steady state behavior of system.

### System Analysis Method

classical Method

or

X<sup>for</sup> function based Method

→ RL

→ PP

→ NP

→ BP

$$U_B + X_A = Y$$

$$U_C + X_D = V$$

#### \* Limitation of classical Method

i) It is valid only for LTI system.  $[X] \times [A] = [X]$

ii) It is valid for SISO (Single Input Single O/p) system.

iii) This Method does not describe the Internal state of the system  
(Initial Value of the System)

iv) This Method gives only Zero state Response.

#### \* Advantages of State Variable Method

i) It is Valid for Linear, Nonlinear, Time Invariant & Time Variant System

ii) It is Valid for both SISO & MIMO System.

and not only in Future Multiple I/p Multiple O/p

\* iii) State Variable Method describes the internal state of the system (Initial Value of the system) step?

\* iv) This Method gives both Zero Input response (ZIR) & Zero State response (ZSR)

\* Representation of State Variable Method

$$\dot{x} = Ax + Bu$$

$$y = Cx + Du$$

$$[\dot{x}]_{nx1} = [A]_{n \times n} [x]_{nx1} + [B]_{n \times n} [u]_{mx1} \quad (1)$$

$$[y]_{px1} = [C]_{pxn} [x]_{nx1} + [D]_{pxn} [u]_{mx1} \quad (2)$$

where,

$$[x]_{nx1} = \text{State Vector} \quad (no. of state variable)$$

State Vector is a collection of state Variable

No. of State Variable  $n :=$  No. of effective Storing element

$=$  No. of time constant

No. of initial value  $\Rightarrow$  Order of  $x$  for function

e.g 1  $n = 1$  ( $x$ )

Date \_\_\_\_\_  
Page \_\_\_\_\_

$$[x]_{1 \times 1} = [x]$$

e.g 2  $n = 2$  ( $x_1, x_2$ )

$$[x]_{2 \times 1} = [x_1]$$

matrix transpose =  $[x]^T$  ①

column Matrix or Vertical Matrix

(Transpose of  $[x_2]$ )  $2 \times 1$  dimension transpose to  $1 \times 2$  - 2

e.g 3  $n = 3$  ( $x_1, x_2, x_3$ )

$$[x]_{3 \times 1} = \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix}$$

1)  $\dot{x}$   $_{n \times 1}$  = Differential State Vector

e.g 4  $n = 2$  ( $x_1, x_2$ )

$$\dot{x} \quad _{2 \times 1} = \begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \end{bmatrix}$$

3)  $[U]_{m \times 1}$  = Input Vector

$m$  = number of Input (or Input Variable)

Input Vector is Collection of Input Variables.

e.g. If  $m = 1$ , no Single Input

$$m = 1 \quad [u] \quad _{1 \times 1}$$

$$[U]_{m \times 1} = [u]_{1 \times 1}$$

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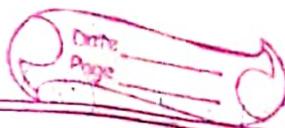
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**Noted-: Single Source Follow, Revise**

**Multiple Time Best key of Success**

eg 2  $m = 2 [u_1, u_2]$  Multiple Input



$$[U] = \begin{bmatrix} u_1 \\ u_2 \end{bmatrix}_{m \times 1}$$

4)  $[Y]_{p \times 1} = \text{output Vector}$

$p = \text{no. of output variables (or simply output)}$

eg 1.  $P = 1$  (Single o/p  $y$ )

$$[Y]_{1 \times 1} = [y]_{p \times 1}$$

{ qstn. question

eg 2  $P = 2$  (Multiple o/p  $y_1, y_2$ )

$$[Y]_{2 \times 1} = \begin{bmatrix} y_1 \\ y_2 \end{bmatrix}_{2 \times 1}$$

{ 1/2 Question

5)  $[A]_{n \times n} = \text{System Matrix (square Matrix)}$

eg  $n = 2$

$$[A]_{2 \times 2} = \begin{bmatrix} A_{11} & A_{12} \\ A_{21} & A_{22} \end{bmatrix}$$

$n = 3$

$$[A]_{3 \times 3} = \begin{bmatrix} A_{11} & A_{12} & A_{13} \\ A_{21} & A_{22} & A_{23} \\ A_{31} & A_{32} & A_{33} \end{bmatrix}_{3 \times 3}$$

⑥  $[B]_{n \times m}$  = Input Matrix

$n = \text{no. of Input Variable}$

$m = \text{no. of state Variable}$

Date \_\_\_\_\_  
Page \_\_\_\_\_

$n \times m$

e.g.  $n=2$   $m=1$

$$[B]_{2 \times 1} = \begin{bmatrix} B_1 \\ B_2 \end{bmatrix}$$

e.g.  $n=2$   $m=2$

$$[B]_{2 \times 2} = \begin{bmatrix} B_{11} & B_{12} \\ B_{21} & B_{22} \end{bmatrix}_{2 \times 2}$$

e.g.  $n=3$   $m=1$

$$[B]_{3 \times 1} = \begin{bmatrix} B_1 \\ B_2 \\ B_3 \end{bmatrix}$$

e.g.  $n=3$   $m=2$

$$[B]_{3 \times 2} = \begin{bmatrix} B_{11} & B_{12} \\ B_{21} & B_{22} \\ B_{31} & B_{32} \end{bmatrix}_{3 \times 2}$$

⑦  $[C]_{p \times n}$  = Output Matrix

e.g.  $n=2$   $p=1$

$$[C]_{1 \times 2} = \begin{bmatrix} C_1 \\ C_2 \end{bmatrix}_{1 \times 2}$$

e.g.  $n=3$   $p=1$

$$[C]_{1 \times 3} = \begin{bmatrix} C_1 \\ C_2 \\ C_3 \end{bmatrix}_{1 \times 3}$$

e.g.  $n=2$   $p=2$

$$[C]_{2 \times 2} = \begin{bmatrix} C_{11} & C_{12} \\ C_{21} & C_{22} \end{bmatrix}_{2 \times 2}$$

⑧  $[E]_{p \times m}$  = data expansion Matrix

↳ no. of input Variable  
↳ no. of output Variable

e.g.  $m=1, p=1$

$$[E]_{1 \times 1} = [0] \quad \left[ \begin{array}{c} 1 \\ 2 \\ 3 \end{array} \right]$$

$m=2, p=2$

$$[E]_{2 \times 2} = \begin{bmatrix} E_{11} & E_{12} \\ E_{21} & E_{22} \end{bmatrix} \quad \left\{ \text{eg. } [E]_{2 \times 2} \right.$$

e.g.  $x = 2 \quad (x_1, x_2)$

$m=1 \quad (y)$

$p=1 \quad (y)$

$$\begin{bmatrix} x_1 \\ x_2 \end{bmatrix}_{2 \times 1} = \begin{bmatrix} A_{11} & A_{12} \\ A_{21} & A_{22} \end{bmatrix}_{2 \times 2} \begin{bmatrix} x_1 \\ x_2 \end{bmatrix}_{2 \times 1} + \begin{bmatrix} B_1 \\ B_2 \end{bmatrix}_{2 \times 1} \begin{bmatrix} u \end{bmatrix}_{1 \times 1}$$

$$\begin{bmatrix} y \end{bmatrix}_{1 \times 1} = \begin{bmatrix} C_1 & C_2 \end{bmatrix}_{1 \times 2} \begin{bmatrix} x_1 \\ x_2 \end{bmatrix}_{2 \times 1} + [E]_{1 \times 1} \begin{bmatrix} v \end{bmatrix}_{1 \times 1}$$

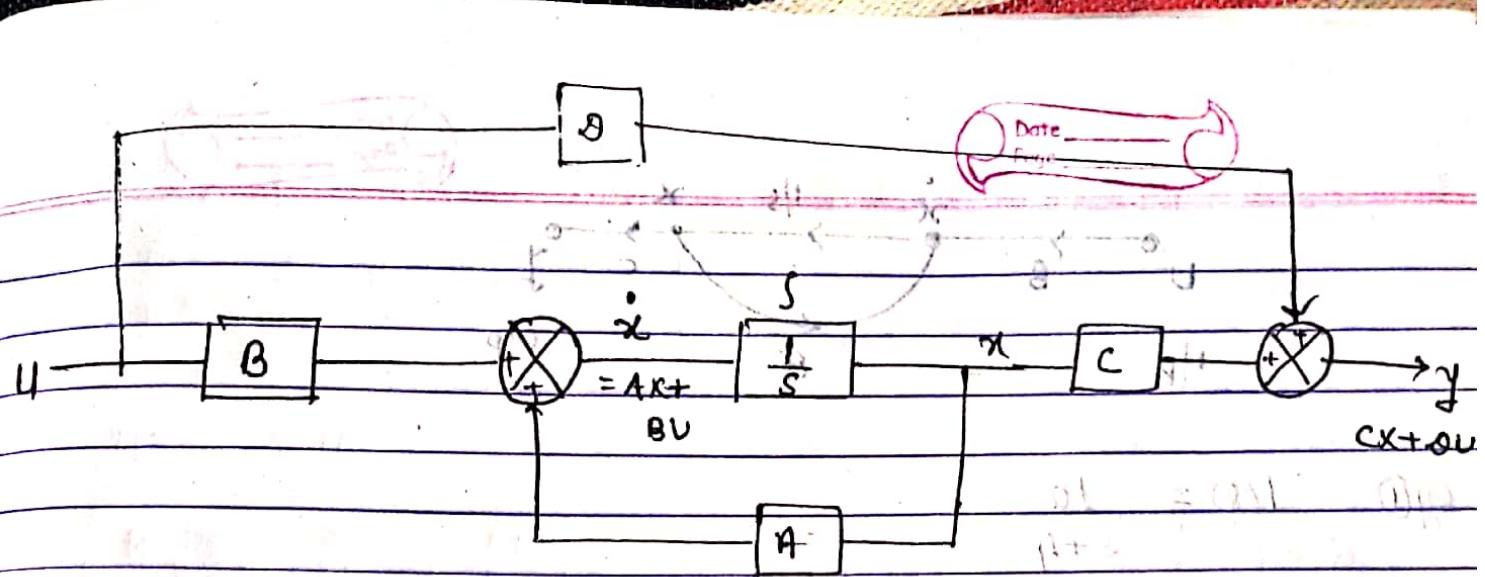
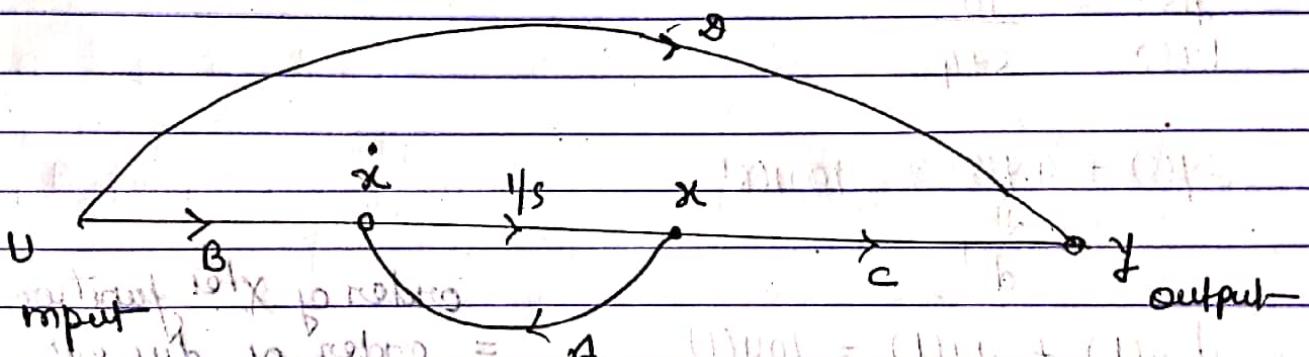


fig: Block Diagram of state variable eqn



additive state fig = Block Diagram of State eqn

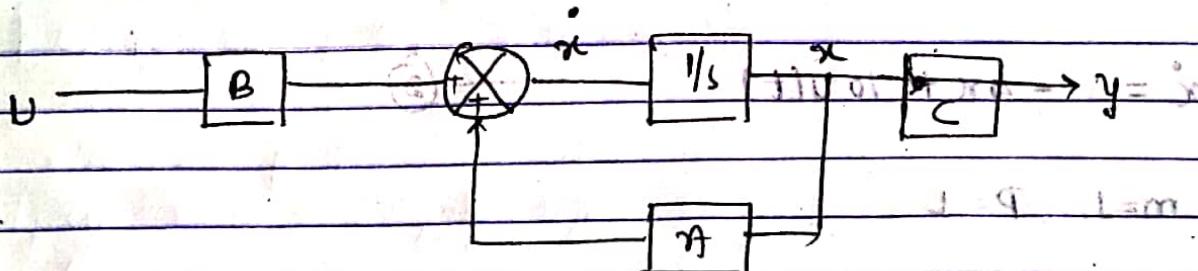
$$y \cdot \theta = 0$$

$$\dot{x} = \frac{(1)u_b}{P_b}$$

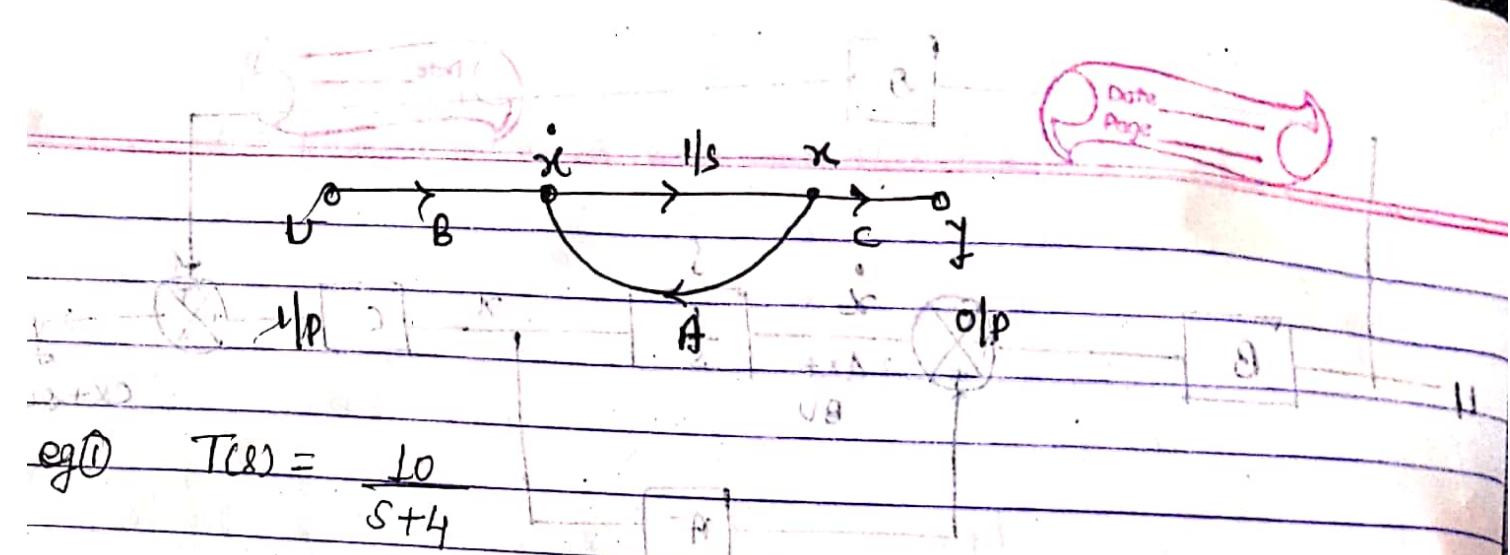
$$\dot{x} = Ax + Bu$$

$$y = Cx$$

$$(1)u_b = xA + \dot{x}$$



$$u[1] \times [a] + u[1] [x] [A] = u[1] [\dot{x}]$$



$$\text{eg(1)} \quad T(s) = \frac{10}{s+4}$$

$$\frac{Y(s)}{U(s)} = \frac{10}{s+4}$$

$$\Rightarrow sY(s) + 4Y(s) = 10U(s)$$

$$\Rightarrow \frac{d}{dt}y(t) + 4y(t) = 10u(t)$$

Let  $y(t) = x$  (state variable)

$$\frac{dy(t)}{dt} = \dot{x}$$

$$\Rightarrow \dot{x} + 4x = 10u(t)$$

$$\dot{x} = -4x + 10u(t)$$

$$n=1, m=1, P=1$$

$$[\dot{x}]_{1x1} = [A]_{1x1} [\dot{x}]_{1x1} + [B]_{1x1} [U]_{1x1}$$

$$[Y]_{1x1} = [C]_{1x1} [\dot{x}]_{1x1} + [D]_{1x1} [U]_{1x1}$$