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Recitation # 3 - Forced Oscillations

Exo # 1

$$m = 0,2 \text{ kg}, \quad \omega_0 = 20 \text{ rad/s}, \quad F_0 = 2 \text{ N}, \quad \gamma = 4 \text{ s}^{-1}$$

a) $\underline{\omega = ?}$

$$\omega = \sqrt{\omega_0^2 - \frac{\gamma^2}{4}} \Rightarrow \omega = \sqrt{\omega_0^2 + \frac{\gamma^2}{4}}$$

N.A: $\omega = 20,10 \text{ rad/s}$

b) $\underline{Q = ?}$

$$Q = \frac{\omega_0}{\gamma} \Rightarrow Q = 5$$

c) $\underline{A(\omega) = ?}$

$$A(\omega) = a \cdot Q = \frac{F_0}{m \omega^2} \cdot Q \quad \text{or} \quad A(\omega) = \frac{F_0}{m \cdot \omega^2 \gamma} \Rightarrow A(\omega) = 0,12 \text{ m}$$

d) $A(\omega_f \ll \omega_0) = a = \frac{F_0}{k} = \frac{F_0}{m \omega_0^2} \Rightarrow A(\omega_f \ll \omega_0) = 0,025 \text{ m}$

Exo # 2

$$\ddot{x} + 4\dot{x} + 8x = 20 \cos(2t)$$

So, we have: $\gamma = 4 \text{ s}^{-1}$, $\omega_0^2 = 8 \text{ rad/s}^2$, $F_0 = 20 \text{ N}$
and $\omega_f = 2 \text{ rad/s}$.

a) the general solution is: $x(t) = x_h(t) + x_p(t)$

$$\Rightarrow x(t) = A_0 e^{-\frac{\gamma}{2}t} \cos(\omega_0 t + \phi) + A(\omega_f) \cos(\omega_f t - \delta)$$

A_0 and ϕ are determined from initial conditions and we have:

$$A(\omega_f) = \frac{F_0/m}{[(\omega_0^2 - \omega_f^2)^2 + \gamma^2 \omega_f^2]^{1/2}} \quad \text{and} \quad \tan(\delta) = \frac{\gamma \omega_f}{(\omega_0^2 - \omega_f^2)}$$

A.N: we get: $A(\omega_f) = 2,24 \text{ m}$, $\omega_f = 2 \text{ rad/s}$ and $\tan(\delta) = 2 \Rightarrow \delta = 63,4^\circ$ or $243,4^\circ$ but $0 < \delta < \pi \Rightarrow \delta = 63,4^\circ \Rightarrow$

$$\omega_f = 2 \text{ rad/s} < \omega_0 = 2,83 \text{ rad/s} \Rightarrow$$

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$$x_G(t) = A_0 e^{-2t} \cos(2t + \phi) + 2,24 \cos(2t - 1,11)$$

$$\dot{x}_G(t) = -2A_0 e^{-2t} \cos(2t + \phi) - 2A_0 \sin(2t + \phi) + 4,48 \sin(2t - 1,11)$$

$$\dot{x}_G(0) = -2A_0 \cos(\phi) - 2A_0 \sin(\phi) + 4,48 \sin(-1,11) = 0$$

$$\begin{cases} x_G(0) = A_0 \cos(\phi) + 2,24 \cos(-1,11) = 0 \quad (1) \\ \dot{x}_G(0) = -2A_0 \cos(\phi) - 2A_0 \sin(\phi) + 4,48 \sin(-1,11) = 0 \quad (2) \end{cases}$$

from (1) and (2) we get $A_0 = 3,15 \text{ m}$, and $\phi = 108,5^\circ \Rightarrow$

$$x_G(t) = 3,15 e^{-2t} \cos(2t + 1,89) + 2,24 \cos(2t - 1,11) \text{ m}$$

b) After a long time \Rightarrow steady state \Rightarrow

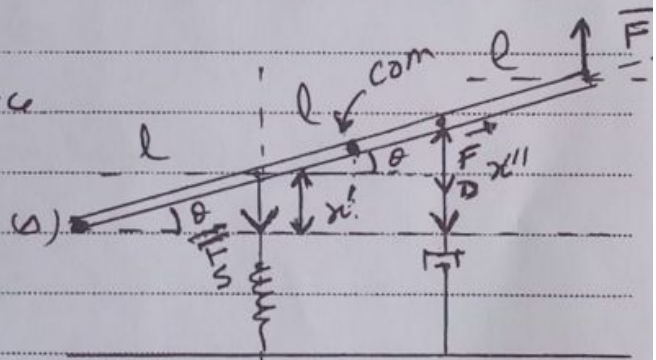
$$x_G(t) \approx x_P(t) = A(\omega_f) \cos(\omega_f t - \delta) = 2,24 \cos(2t - 1,11)$$

$$\Rightarrow T_f = \frac{2\pi}{\omega_f} = \pi \text{ s and } f = \frac{2\pi}{T_f} = 0,32 \text{ Hz}, A(\omega_f) = 2,24 \text{ m}$$

EX0 #3

\vec{F} : spring, \vec{F}_D : damping force

$$\sum \vec{\tau} = I_P \ddot{\theta} \Rightarrow \vec{\tau}_s + \vec{\tau}_D + \vec{\tau}_F = I_P \ddot{\theta}$$



$$\vec{\tau}_s = \vec{R}_s \wedge \vec{F} \Rightarrow \tau_s = -l k x' \sin\left(\frac{\pi}{2} + \theta\right) = -l k x' \cos(\theta)$$

$$\vec{\tau}_D = \vec{R}_D \wedge \vec{F}_D \Rightarrow \tau_D = -2l \cdot b \cdot \dot{x}' \sin\left(\frac{\pi}{2} + \theta\right) = -2l \cdot b \cdot \dot{x}' \cos(\theta)$$

$$\vec{\tau}_F = \vec{R}_F \wedge \vec{F} \Rightarrow \tau_F = +3l \cdot F \cdot \sin\left(\frac{\pi}{2} + \theta\right) = +3l \cdot F \cdot \cos(\theta)$$

for small oscillations $\Rightarrow \cos(\theta) \approx 1$ and $\sin(\theta) \approx \frac{x'}{l} = \frac{\dot{x}'}{2l} = \dot{\theta}$

$$\Rightarrow \frac{\ddot{x}'}{2l} = \dot{\theta} \Rightarrow \ddot{x}' = 2l \ddot{\theta}$$

we replace in our diff-equ, we get:

$$-l k (l\theta) - 2l b (2l\dot{\theta}) + 3l F = m (3l) \ddot{\theta}$$

where $I_D = m D^2 = m (3l)^2$

$$\ddot{\theta} + \frac{4}{9} \frac{b}{m} \dot{\theta} + \frac{1}{9} \frac{k}{m} \theta = \frac{F_0}{3lm} \cos(\omega_f t)$$

$$\ddot{\theta} + \gamma \dot{\theta} + \omega_0^2 \theta = \frac{F_0}{3lm} \cos(\omega_f t) \quad \left| \begin{array}{l} \gamma = \frac{4}{9} \frac{b}{m} \text{ and} \\ \omega_0 = \sqrt{\frac{1}{9} \frac{k}{m}} \end{array} \right.$$

N.A: $\gamma = 0,44 \text{ s}^{-1}$, $\omega_0 = 2,98 \text{ rad/s}$
 $\omega_f = 2\pi \text{ rad/s}$

\Rightarrow The steady state response is $\theta(t) = \theta(\omega_f) \cos(\omega_f t - \delta)$

$$\theta(\omega_f) = \frac{F_0 / 3ml}{[(\omega_0^2 - \omega_f^2)^2 + \gamma^2 \omega_f^2]^{\frac{1}{2}}} = 0,44 \text{ rad}$$

$$\tan(\delta) = \frac{\gamma \omega_f}{(\omega_0^2 - \omega_f^2)} \rightarrow \delta = 5,16^\circ \text{ or } 174,84^\circ \text{ but we have } \omega_f > \omega_0 \Rightarrow \delta = 174,84^\circ$$

$$\theta(t) = 0,44 \cos(2\pi t - 3,05) \text{ rad}$$

EXOT 4 :

$m = 2 \text{ kg}$, $\Delta x_0 = 2,50 \text{ cm}$, $a = 1,00 \text{ m}$, $Q = 15,0$
a) $\omega = ?$ $\omega_0 = \sqrt{\frac{k}{m}}$

at equilibrium $\sum \vec{F} = 0 \Rightarrow \vec{F}_s + \vec{W} = 0 \Rightarrow -k \Delta x_0 + mg = 0$

$$\Rightarrow \left(k = \frac{mg}{\Delta x_0} = 784 \text{ N.m}^{-1} \right) \Rightarrow \left(\omega_0 = 19,8 \text{ rad/s} \right)$$

b) The amplitude:

at resonance: $A(\omega) = a Q = 10^{-3} \times 15 \Rightarrow A(\omega) = 1,5 \text{ cm}$

c) The power:

$P(\omega_f) / \omega_f = \omega_0 + 2\% \omega_0 = \omega_0 + 0,02 \omega_0 = 20,2 \text{ rad/s}$
and to

$$P(\omega_f) = 0,086 \text{ W}$$

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EX 04

$$m = 0,03 \text{ kg}, \quad F_0 = 0,12 \text{ N}, \quad \omega_f = 20 \text{ rad/s}, \quad b = 0,06 \text{ kg/s}$$

$$x(t) = A \sin(20t)$$

after a certain time $x(t) = A(\omega_f) \cdot \cos(\omega_f t - \delta)$

$$\Rightarrow x(t) = A(\omega) \cdot \cos(20t - \frac{\pi}{2})$$

a) $A(\omega) = ?$: from the eqn of displacement $x(t)$ we get that $\omega_f = 20 \text{ rad/s}$ and $\delta = \frac{\pi}{2}$, so \Rightarrow the system is at resonance because $\delta = \frac{\pi}{2}$

$$\Rightarrow A(\omega = \omega_0) = a \cdot Q = a \cdot \frac{\omega_0}{\gamma} = \frac{F_0}{k} \cdot \frac{\omega_0}{\gamma} = \frac{F_0}{m} \cdot \frac{1}{\gamma \omega_0}$$

$$A(\omega) = 10 \text{ cm}$$

b) $E = ?$ $E(t) = \frac{1}{2} k x^2 + \frac{1}{2} m \dot{x}^2$
 $= \frac{1}{2} k A^2(\omega_f) \cos^2(20t - \frac{\pi}{2}) + \frac{1}{2} m A^2(\omega_f) \omega_f^2 \sin^2(20t - \frac{\pi}{2})$

$$E(t) = \frac{1}{2} A^2(\omega) \cdot m \left[\omega_0^2 \cos^2(20t - \frac{\pi}{2}) + \omega_f^2 \sin^2(20t - \frac{\pi}{2}) \right]$$

we are in the case of resonance $\Rightarrow \omega_f = \omega_0 \Rightarrow$

$$E(t) = \frac{1}{2} m A^2(\omega_0) \omega_0^2 = 6 \cdot 10^{-2} \text{ J} \rightarrow \text{stored energy}$$

c) $\bar{E}_{\text{dis}} = ?$ $\bar{P} = \frac{1}{T} \int_0^T P \cdot dt$ $P_{\text{dis}} = -b \dot{x}^2 \Rightarrow$

$$\bar{P}_{\text{dis}} = -\frac{b A^2(\omega) \cdot \omega^2}{2} \int_0^T \sin^2(20t - \frac{\pi}{2}) dt = -\frac{b A^2(\omega) \cdot \omega^2}{2} \cdot \frac{T}{2}$$

$$\bar{P}_{\text{dis}} = -\frac{b A^2(\omega_0) \cdot \omega_0^2}{2} = -0,12 \text{ W}$$

or we use the expression of $\bar{P}(\omega) = \frac{F_0^2}{2m} \frac{\gamma \omega_f^2}{[(\omega_0^2 - \omega_f^2)^2 + \gamma^2 \omega_f^2]}$

at resonance $\bar{P}(\omega) = \frac{F_0^2}{2m} \cdot \frac{F_0^2}{\gamma} = 0,12 \text{ W}$

$$\Rightarrow \bar{P}_{\text{dis}}(\omega) = -0,12 \text{ W} \Rightarrow \bar{P}_{\text{dis}} = \frac{\bar{E}_{\text{dis}}}{T} \Rightarrow$$

$$\bar{E}_{\text{dis}} = \bar{P} \cdot \frac{2\pi}{\omega_0} = -0,038 \text{ J} \quad (\text{at resonance})$$

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d) $\overline{E}_{di} = ?$ we have $\overline{E}_{di} = \overline{P}_{di} \cdot T \Rightarrow$

$$\overline{E}_{di} = \frac{b A(\omega_0) \cdot \omega_0^2}{2} \cdot \frac{2\pi}{\omega_0} = b A(\omega_0) \cdot \omega_0 \pi$$

$$E = \frac{1}{2} A(\omega_0) \cdot \omega_0^2 \cdot m \Rightarrow$$

$$\frac{\overline{E}_{di}}{\overline{E}_{st}} = \frac{2b}{m} \cdot \frac{\pi}{\omega_0} = 2\pi \cdot \frac{\gamma}{\omega_0} = \frac{2\pi}{Q} = 0, 2\pi$$

Exo 6

\Rightarrow lower frequencies $\Rightarrow \omega < \omega_0 \Rightarrow A(\omega) = a = 0,01 \text{ mm}$
 At resonance (because the experiment is done for resonance)
 $\Rightarrow \omega_f \approx \omega_0 \Rightarrow A(\omega_0) = aQ = 5 \text{ mm}$ and so;

$$f_0 = 250 \text{ Hz} \Rightarrow \omega_0 = 2\pi f_0 = 500\pi \text{ rad/s}$$

a) $Q := ?$ $A(\omega_0) = aQ = 5 \text{ mm} \Rightarrow Q = 500$

b) $BW = ?$ $BW = \gamma = \frac{\omega_0}{Q} = \frac{500\pi}{500} \Rightarrow BW = \pi \text{ s}^{-1}$

Exo 7

$$V(t) = ?$$

Kirchhoff's law for current:

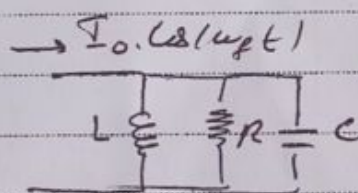
$$\frac{I}{L} + \frac{I}{R} + \frac{I}{C} = I(t) = I_0 \cos(\omega_f t)$$

$$V_L = L \frac{dI_L}{dt} \Rightarrow \frac{I}{L} = \frac{1}{L} \int_0^t V_L dt$$

$$\frac{V}{R} = R \frac{I}{R} \Rightarrow \frac{I}{R} = \frac{V}{R}$$

$$\frac{V}{C} = \frac{q}{C} \Rightarrow \frac{I}{C} = \frac{dq}{dt} = C \frac{d(q/C)}{dt} = C \frac{d(V/C)}{dt}$$

We replace in our equation, we get:



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$$\frac{1}{L} \cdot \int V_L dt + \frac{V_R}{R} + C \frac{d}{dt} V_C = \frac{I_0}{\omega} \cos(\omega t) \quad (*)$$

$V_R = V_C = V_L = V$ (parallel circuit), we differentiate the eqn (*) we get

$$\frac{1}{L} V + \frac{1}{R} \frac{dV}{dt} + C \frac{d^2 V}{dt^2} = -\frac{I_0}{\omega} \sin(\omega t)$$

$$\Rightarrow C \ddot{V} + \frac{1}{R} \dot{V} + \frac{1}{L} V = -\frac{I_0}{\omega} \sin(\omega t)$$

$$\ddot{V} + \frac{1}{RC} \dot{V} + \frac{1}{LC} V = -\frac{I_0}{C} \cdot \frac{\omega}{f} \sin(\omega t)$$

b). from the diff. eqn, we have:

$$\gamma = \frac{1}{RC}, \quad \omega_0 = \frac{1}{\sqrt{LC}}, \quad \frac{F_0}{m} \equiv Q = \frac{\omega_0}{\gamma} \Rightarrow$$

$$Q = \frac{1}{\sqrt{LC}} \cdot RC = R \sqrt{\frac{C}{L}}$$

$$BW = \gamma = \frac{1}{RC}$$

$$c) \bar{P}(\omega) = ?$$

$$\bar{P}(\omega) = \frac{1}{2m} \frac{F_0^2 \gamma \omega^2}{[(\omega_0^2 - \omega^2)^2 + \gamma^2 \omega^2]} \quad \text{in mechanical system.}$$

$$\bar{P}_{mech}(\omega) = \frac{1}{2m} \cdot \frac{F_0^2}{\gamma} \quad \text{by identification with electrical system.}$$

$$F_0 \equiv I_0, \quad m \equiv C, \quad k = \frac{1}{L}, \quad b \equiv \frac{1}{R} \Rightarrow$$

$$\bar{P}_{mech}(\omega) = \frac{I_0^2}{2C} \cdot R \cdot C \Rightarrow \bar{P}_{mech}(\omega) = \frac{1}{2} I_0^2 R$$