

* Basics & intro to control.

* Any system which is not having sensor and not having provision to select the Reference i/p. Is open loop system

Having sensor and provision to select Reference i/p is close loop system.

* Fan, tubelights, air-cooler, automatic washing machine, traffic lights & so on are open loop system.

* AC, Refrigerator, human being, automatic door box, gizmo are closed loop system.

* Steady state error valid only for unity feedback system if non unity feedback is given to calculate steady state error we require to convert into unity feedback.

$$* T.F = \frac{L(O/P)}{L(I/P)} \Big|_{Z_i=0} = \underline{L(IR)} \quad \text{transfer fn analysis initial condition must be zero.}$$

* impulse Response also called as system response, natural response or free forced response.

* Basically system described in the form of open loop transfer function. $\frac{C(s)}{R(s)} = G_1(s) = K(1+s\tau_1)(1+s\tau_2)\dots \rightarrow \text{standard form.}$

τ \rightarrow time constant order \Rightarrow maximum power in the denominator

type \Rightarrow No. of poles at origin i.e. total no. of poles in s-plane

* To find value of K convert the given f^n in standard form.

* Type and order of close loop system defined for open loop transfer function of unity feedback system.

\therefore To get type & order of close loop system required $G(s)$ $M(j\omega)$ should equal to 1.

* $M(j\omega)$ \rightarrow magnitude of $G(j\omega)$

* denominator of T.F. equate to zero becomes characteristic equation.

For close loop system characteristic eqn is $1 + G(s)H(s) = 0$

* Pole and zeros are nothing but v-e of inverse of system timeconstant ($-\frac{1}{T_p}$) from standard eqn.

pole ($-\frac{1}{T_p}$) at which magnitude of T.F. is 00

zeros ($-\frac{1}{T_p}$) at which magnitude of T.F. is 0.

* Time constant gives behaviour of system.

γ large. \Rightarrow slowest system Response.

\Rightarrow large time to take steady state.

* Dominant pole :- pole which is close to imaginary axis.

Insignificant pole :- lies left most. $T_{SP}(\gamma) \leq D_P(\gamma)$

$T_{SP} \leq 5$ times D.P.

* T_{SP} is best pole. \because very quick Response & more stable.
because of DP system Response becomes slow & less stable, if ISP neglected no much change in system Response

$$L[e^{at}] = \frac{1}{s-a}$$

* exponential power gives real part of poles

$$L[\sin \omega_n t e^{bt}] = \frac{b+is}{s^2+b^2}$$

* sin & cosine terms give imaginary part of pole

$$L[e^{at} \sin bt]$$

* other terms represent repeated nature of poles.

$$L[t^n e^{-at}]$$

n+1 poles repeated

* To find equivalent system by neglecting JSP first do T.F. in standard form then value of $T \ll k$ then neglect.

* finding system Response, system stability, system time constant we consider only poles but not zeros.

open loop zeros not affect open loop stability.

close loop zero not affect close loop stability.

but open loop zeros affect close loop stability.

* → movement of pole in the s-plane is nothing but varying the system component values.

→ absolute stability \Rightarrow stable for all values of system component i.e. k from 0 to ∞ .

→ conditional stable \Rightarrow stable for certain range of system component.

→ addition pole or zero to TF means adding RLC component.

* Real part of pole gives Time constant & imaginary part of pole gives frequency.

* Non Repeated poles on jw axis give constant amplitude & frequency Response undamped system. system is marginally stable

* Pole lies on left side of s-plane decaying amplitude & oscillations damped system system is stable.

* poles lie on Right side of s-plane amplitude & oscillation exponentially increase over damped & unstable.

* Transfer fn of opamp.

$$\text{Inverting mode} \Rightarrow \frac{V_o(s)}{V_i(s)} = -\frac{Z_2(s)}{Z_1(s)}$$

$$\text{Non-inverting mode} \Rightarrow \frac{V_o(s)}{V_i(s)} = \left[1 + \frac{Z_2(s)}{Z_1(s)} \right]$$

* T.F. to the diff. equations.

$$\frac{d^n y}{dt^n} = s^n y(s)$$

T.F. is not defined for Non-linear System.

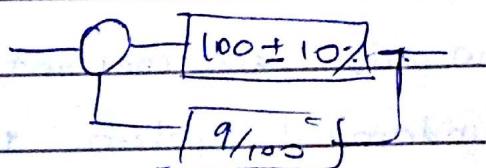
Sensitivity: - relative variation in opa to the change in parameters ① $G(s)$ ② $H(s)$

$$S^T_H = \frac{\% \text{ of change in T.F.}}{\% \text{ of change in } G(s)} = \frac{\partial T/F}{\partial G/H}$$

$$\text{By } S^T_H = \frac{\partial T/F}{\partial H/H}$$

for open loop system $T=G$ $S^T_H = 1$ highly sensitive

for closed loop system $T = \frac{G}{1+GH}$ $S^T_H = \frac{1}{1+GH}$ less sensitive than OLS.

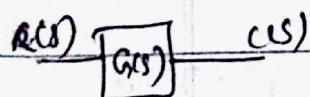


Now by $S^T_H = \frac{\% \text{ change in } T}{\% \text{ change in } H} = \frac{1}{1+GH}$ calculate from given data

* block diagram.

* Open loop :- * $T \cdot F = \frac{C(s)}{R(s)} = G(s)$ O.L.T.F.

* gain is same not affected.



* more stable

* less accurate

* more sensitive w.r.t. noise & disturbance.

* B.W. is constant \because Gain \times Bandwidth = constant.

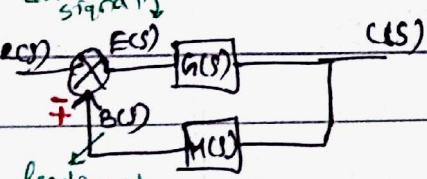
* more reliable.

* Not necessary to measure o/p errors. are not generated.

* sensors are not essential

* design easy.

* close loop :- * forward path gain $G(s) = \frac{C(s)}{E(s)}$



backward path gain $H(s) = \frac{B(s)}{C(s)}$

$[G(s)H(s)] = \text{loop gain}$

+ve feed back 0 or $\pm 360^\circ$ phase shift

-ve feed back $\pm 180^\circ$ phase shift.

$$\frac{C(s)}{R(s)} = \frac{G(s)}{1 \pm G(s)H(s)}$$

* gain is decreased by factor $1 + GH$

* stability depends on loop gain.

Loop gain = 0 \Rightarrow O.L. stability = L stability.

$G(s)H(s)$ O.L.T.F of N.U.F. system

* Accuracy depends on feed back N/W.

$H(s) = 1$ O.L.T.F. of U.F. system.

Much more accurate than O.L. system.

* less sensitive w.r.t. noise & disturbance.

* gain is reduced by $1 + GH$ so B.W. is increased by $H/(1+GH)$ factor.

* less reliable

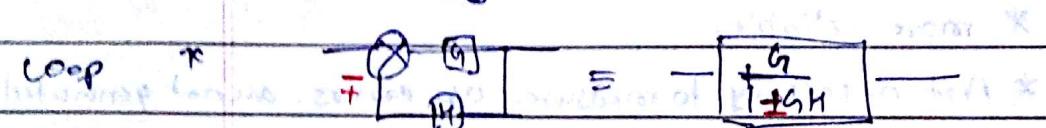
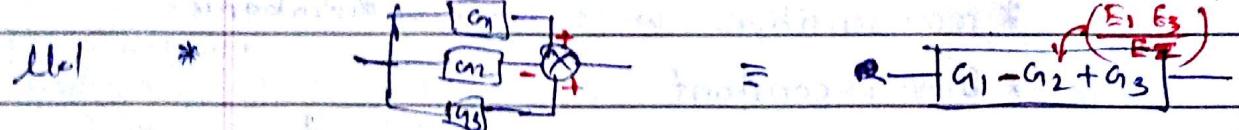
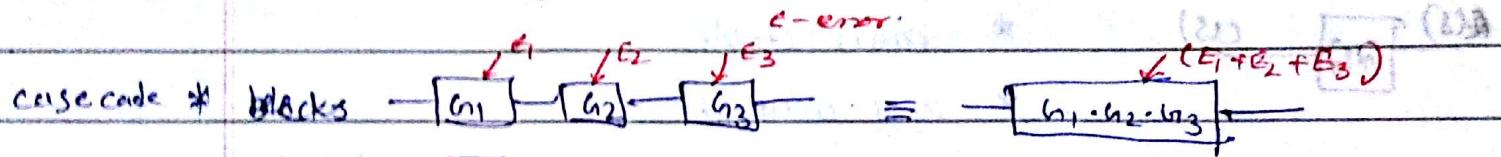
* o/p must be measured errors are generated.

* sensors are essential ~~design is complex~~

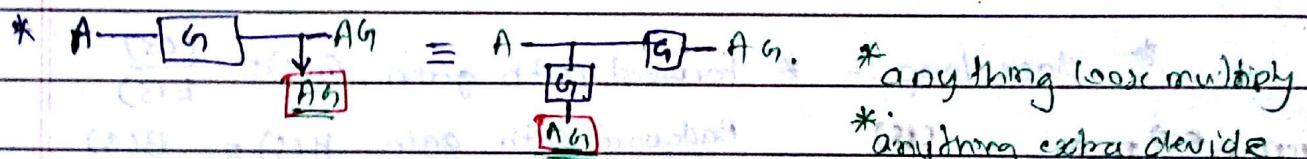
* design is complex

* Block diagram Reduction techniques.

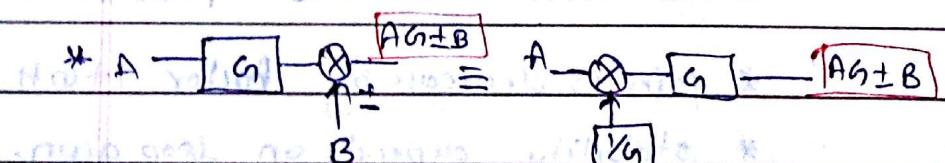
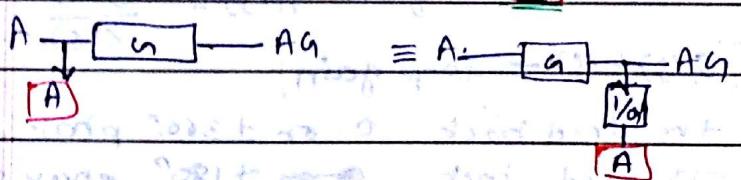
- * Imp. rule is gain should remain same before shifting & after shifting. maintain o/p same at each termination.



Summary * No change but block should not be there in b/w two S.P.

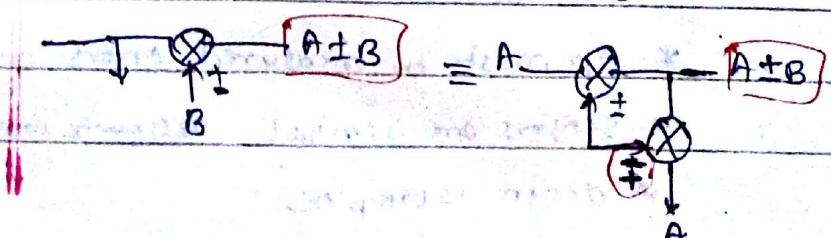
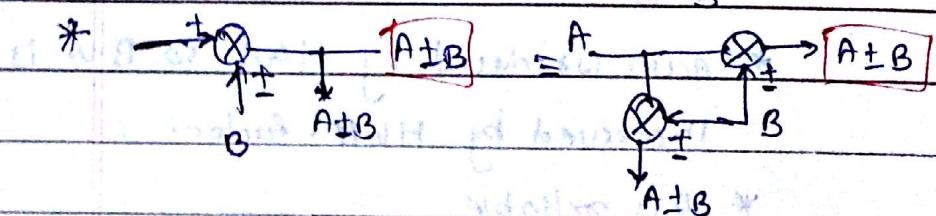
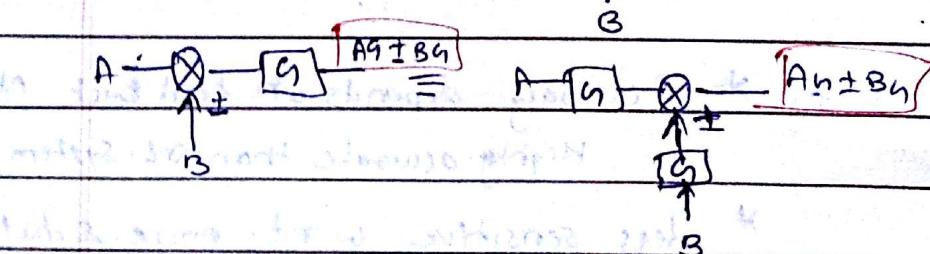


* anything extra divide



Cond to be follow

(\square → before shifting equals to \square → after shifting)



$$N = \frac{D}{D + N \cdot H} = \frac{b}{1 + bH}$$

- * while shifting sign take care of forward path gain before shifting it should not be changed. so if forward path is changing, eliminate that factor by multiplying or dividing that block in additional path.
- * if no H.B is mention consider open loop response.

Mason's gain formula:-

$$M = \frac{Y_{out}}{Y_{in}} = \sum_{k=1}^N \frac{M_k \Delta_k}{\Delta}$$

$$M = \text{gain betn } Y_{in} \& Y_{out}$$

Y_{out} = O/p node variable.

Y_{in} = Input node variable.

N = total number of forward path betn Y_{in} & Y_{out} .

M_k = gain of the k^{th} forward path betn Y_{in} & Y_{out} .

$\Delta = 1 - (\text{sum of all individual loop gains}) + (\text{sum of gain of all possible combination of two non-touching loops})$

$\Delta_{(min)}$ (sum of the gain products of all possible comb.)

of three non-touching loops + ... ,

Δ_k is obtained from Δ by removing the loops touching min fw. path.

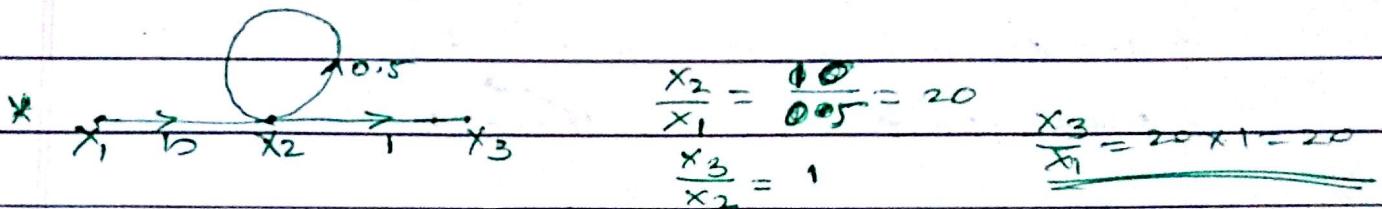
* Signal flow Graph (SFG) applied only for linear system.

* SFG may be defined as a graphical means of portraying the i/p - o/p relationship between the variables of a set of linear algebraic equations.

* Block diagrams contains system o/p variables, system i/p variables & functional relation of the variables.

* in SFG. the nodes represent system variables.

Branches of SFG represent functional relations of the variables.



* O/P of FB control system must be a fⁿ of reference i/p & error signal.

* AC control system has the advantage of smaller frame size & AC components.

* T-F. of control system depends on system parameters only.

* Potentiometers are used in control system as error sensing transducers.

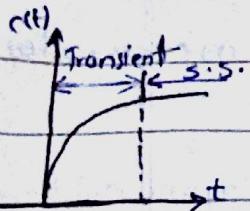
* T.F. of Tachometer is K_s.

* If BD is complicated then go by Mason's gain formula.

* if transportation lag $T_d = a$ sec

$$\text{then } L[\text{transport lag } T_d] = e^{-as}$$

* Time domain Analysis.



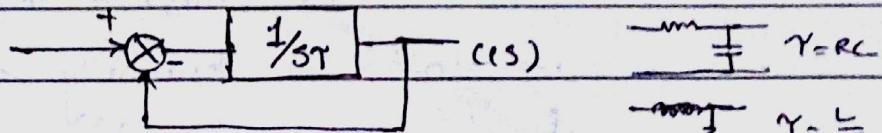
$$T_R \cdot c(t) = C_t r(t) + C_{ss}(t)$$

* exponential decay terms gives transient response.
i.e. poles in the left side gives transient response.

* Poles in the imaginary axis gives steady state response.

$e^{-\alpha t}$ terms are transient & constant or sine cosine are steady state.

* 1st order system.



$$* G(s) = \frac{1}{\tau s} \quad H(s) = 1$$

$$CLTF = \frac{1}{(1+s\gamma)} \quad \text{Type=1 order=1 (OL)}$$

Type & order for close.L not defined

* Error is deviation of o/p from i/p $e(t) = r(t) - c(t)$

$$\text{Steady state error} = \lim_{t \rightarrow \infty} e(t) [r(t) - c(t)] = e_{ss}$$

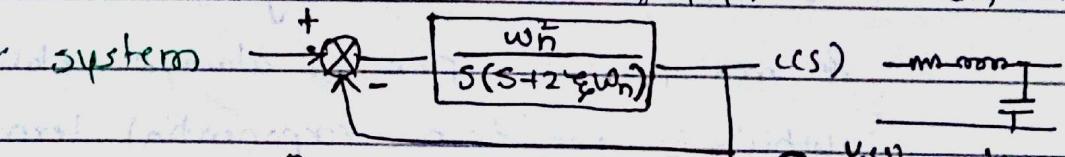
* to find various Response (e.g. Impulse response, step response etc)

take $R(s)$ (e.g. for impulse $R(s)=1$, for step $R(s)=\frac{1}{s}$) multiply
to T.F. & get $C(s)$ by ILT.

* for sinusoidal Response i/p. sinusoidal o/p also sinusoidal.

$$r(t) = A \sin(\omega t + \phi) \Rightarrow c(t) = A \cdot M \sin(\omega t + \phi + \phi') \quad \begin{matrix} \downarrow \\ \text{magnitude} \\ \text{of T.F.} \end{matrix} \quad \begin{matrix} \downarrow \\ \text{angle of T.F.} \end{matrix}$$

* 2nd order system



$$① \frac{C(s)}{R(s)} = \frac{w_n^2}{s^2 + 2\xi w_n s + w_n^2}$$

$$② \frac{V_o(s)}{V_i(s)} = \frac{\frac{1}{s}}{s^2 + s(\frac{R}{L}) + \frac{1}{LC}}$$

Comparing ① & ②.

$$w_n = \frac{1}{\sqrt{LC}} \quad \text{rad/sec} \quad \xi = \frac{R}{2\sqrt{LC}} \quad \text{damping ratio}$$

ξw_n = Damping factor / actual damping. $\frac{\text{ratio of energy lost}}{\text{to energy stored.}}$

* 2nd order system Response depends on ξ only. (i.e. $\xi > 0$)

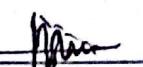
* 2nd order system is stable for all the +ve values of ξ
if poles lies in left of s-plane. $\xi < 0$

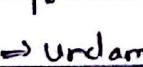
* clamped, undamped, overdamped etc. graphs are generated from terms in (t) .

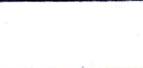
basically two terms are there ① transient \Rightarrow exponential term

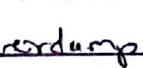
② steady state \Rightarrow sinusoidal oscillation term.

if only e^{-at} (i.e.) decaying term \Rightarrow  overdamped

if e^{-at} with sinusoidal term then \Rightarrow  damped oscillation,

if sinusoidal term only \Rightarrow  undamped oscillation.

if e^{+at} (rising exponential) \Rightarrow 

if e^{+at} with sinusoidal \Rightarrow  overdamped.

* ζ is damping ratio which gives damping characteristics & pole locations.

~~cc~~ * * * T (time constant) it is ∞ at $s=0$ $\therefore s=\frac{1}{T}$

if $T \rightarrow \infty$ (i.e. pole nearer to jw axis) i.e. $s \rightarrow 0$

System Response is very slow. ($\zeta = 0$) on jw axis

\therefore when $\zeta = 0$ $T = \infty$, $s = 0$ system response is slowest.

system is marginally stable with undamped oscillations

since poles on jw axis. value of pole on jw axis

i.e. imaginary value gives oscillation frequency.

& real part of pole is always related to time constant which is get from exponential term.

\therefore exponential term gives real part i.e. time constant

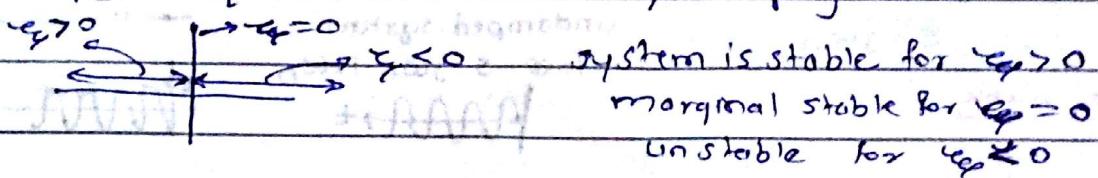
\therefore sinusoidal term gives imaginary part i.e. freq. of oscillations

as poles ~~approach~~ $\rightarrow \infty$, $T \rightarrow 0$ system is fast & more stable

($\zeta > 1$)

Now for 2nd order system damping is depending on ξ value
following points are important

- * ~~worst damping condition~~ $\rightarrow \xi = 0$ ~~stable marginally~~ $\rightarrow \xi > 0$ \rightarrow damping ratio.



- * When $\xi = 1$ $\underline{s = -\omega_n}$ pole is left side $-\omega_n \therefore T = \frac{1}{\omega_n}$

$\xi = 0 \rightarrow \omega_d = \omega_n$ pole on jw axis ω_n freq. of oscillation

- * When $\xi \uparrow$ from -1 to 1 2nd order pole path is a circle with radius ω_n (ξ value gives pole location)

- * $\xi \uparrow$ from 0 to 1 2nd order pole moves from jw axis towards left of real axis and through a path of radius ω_n \therefore

$\therefore T \downarrow$ sec., settling time \downarrow sec., $\omega_d \uparrow$ sec. \therefore damped oscillation \downarrow sec.
as ω_d decreases $t_r, t_d, t_p \uparrow$ sec., system becomes fast relatively
& relative stability \uparrow sec.

- * When $\xi \geq 1$ & increases. then we get only real axis poles.

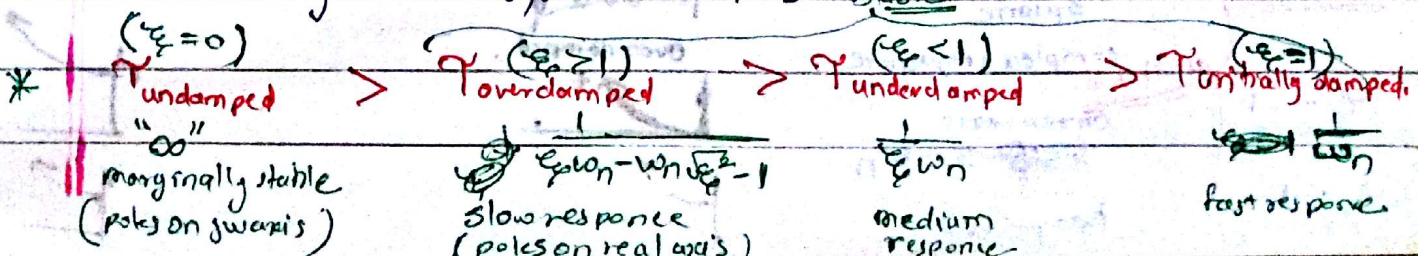
i.e. two poles on real axis. ~~one pole is more towards origin and another pole is more towards infinity~~

$\xi = 1 \quad s = -\omega_n$ as $\xi \uparrow$ from 1 \rightarrow one pole is more towards origin and another pole is more towards ∞ .

the pole which is moving towards ∞ is ISP \therefore can neglected since it won't affect the system but the pole moving toward origin is dominant and having following effects

\therefore as $\xi \geq 1$ $T \uparrow$ sec. \therefore settling time \uparrow sec. ω_d becomes 0 \therefore poles on real axis

\therefore stability of the system is \downarrow sec. \therefore stable



ξ Value Pole location Impulse Response Unit Step Response

$\xi = 0$

imaginary axis

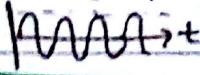
$$s = j\omega_n$$

marginal stable

$$c(t) = \omega_n \sin \omega_n t$$

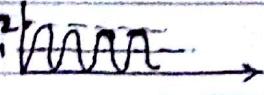
undamped system

$$T = \infty \quad s = j\omega_n \quad f = \omega_n$$



$$c(t) = 1 - \cos \omega_n t$$

\rightarrow " "



$0 < \xi < 1$

left side of

s plane

having both
real & imag. part.

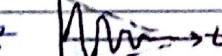
i.e. complex pole

$$s = -\xi \omega_n + j\omega_n \sqrt{1-\xi^2}$$

$$c(t) = \frac{\omega_n}{\sqrt{1-\xi^2}} \sin(\omega_n \sqrt{1-\xi^2} t) \cdot e^{-\xi \omega_n t}$$

damped or
underdamped.

$$c(t) = 1 - \frac{e^{-\xi \omega_n t}}{\sqrt{1-\xi^2}} \sin \left[\omega_n \sqrt{1-\xi^2} t + \tan^{-1} \left(\frac{\sqrt{1-\xi^2}}{\xi} \right) \right]$$



$$\gamma = \frac{1}{\xi \omega_n} \quad f = \omega_d = \omega_n \sqrt{1-\xi^2}$$

stable

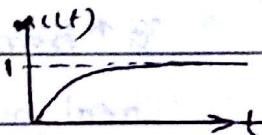
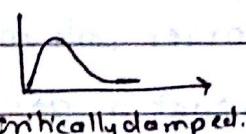
$\xi \geq 1$

$$s = -\omega_n \text{ on}$$

real axis

$$c(t) = \omega_n^2 t e^{-\omega_n t}$$

$$(t) = 1 - e^{-\omega_n t} [\omega_n t + 1]$$



~~theorems~~

stable

$$\gamma = \frac{1}{\omega_n} \quad f = \omega_d = 0$$

$\xi > 1$

poles on real axis.

~~oscillation~~

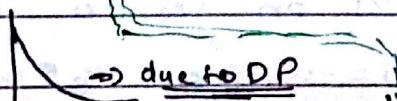
$$s = -\xi \omega_n \pm \omega_n \sqrt{\xi^2 - 1}$$

$$c(t) = K_1 e^{-(\xi \omega_n - \omega_n \sqrt{\xi^2 - 1})t} - K_2 e^{-(\xi \omega_n + \omega_n \sqrt{\xi^2 - 1})t}$$

D.P. → affect response

ISP

$$c(t) = 1 - K_1 e^{-(\xi \omega_n - \omega_n \sqrt{\xi^2 - 1})t} - K_2 e^{-(\xi \omega_n + \omega_n \sqrt{\xi^2 - 1})t}$$

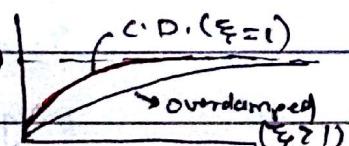


overdamped, system

$$\gamma = \frac{1}{\sqrt{\omega_n^2 + \omega_n^2 \sqrt{\xi^2 - 1}}}$$

minus due to D.P near ω_n

$$F = 0 \text{ rad/sec.}$$



C.D. ($\xi = 1$)

overdamped

($\xi^2 > 1$)

~~other cases~~ Rest cases on RHS compare with above three cases for γ & f but they are unstable

$-1 < \xi < 0$

Poles on L.H.S. of
s plane

complex conjugate

on real axis

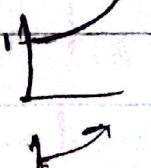
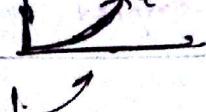
$$s = \omega_n$$

$\xi \geq 1$

two poles.



overdamped.



* time domain specification.

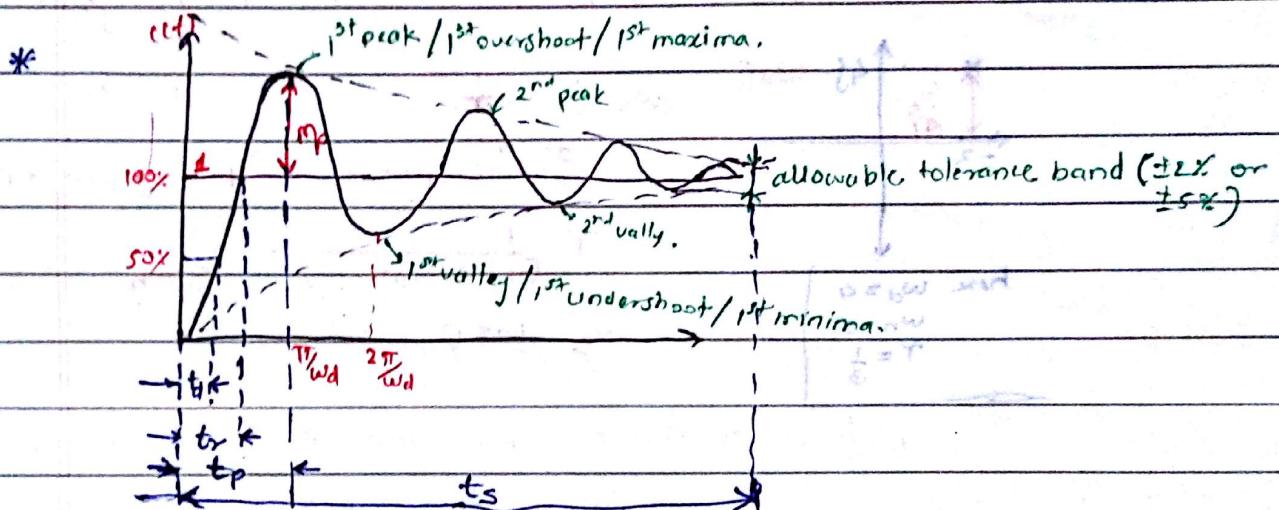
* for time domain specification we use unit step i/p

	transient	steady state	stability
impulse	✓	✗	practically not exist
Step	✓	✓	exist
ramp	✓	✓	X } unbounded & unstable
parabolic	✓	✓	X }

* Underdamped system with ξ , 0.4 to 0.7 is best for time

domain analysis, because it is stable, having less tr (rise time) & less settling time (t_s)

$$c(t) = 1 - e^{-\zeta \omega_n t} \sin [\omega_n \sqrt{1-\zeta^2} + \tan^{-1}(\frac{\sqrt{1-\zeta^2}}{\zeta})] t$$



* delay time \rightarrow 0 to 50% $t_d = \left(\frac{1+0.7\xi}{\omega_n} \right)$ sec.

* Rise time \rightarrow 10 to 90% $t_r = \pi - \tan^{-1} \left(\frac{\sqrt{1-\xi^2}}{\xi} \right)$ sec.

$$t_r = \frac{\pi}{\omega_n} \cos^{-1} \xi \text{ sec.}$$

* Peak Time \rightarrow 0 to peak at time $t_p = \frac{n\pi}{\omega_n}$ sec. $n=1$ for 1st overshoot.

* Overshoot \rightarrow diff. Btⁿ time response at peak $m_p = [c(t_p) - c(\infty)]$ to steady state value.

* % Peak overshoot \rightarrow normalise diff. betw $m_p = \left[\frac{c(t_p) - c(\infty)}{c(\infty)} \right] \times 100\%$
 $c(\infty)$ for unit step i/p = 1

* for 1st peak $n=1$ for 1st overshoot, $n=3$ for 3rd overshoot
 $(\text{for 1st valley } n=2) (\text{2nd valley } n=4)$ $\pm 5\% \quad t_s = 3\gamma \quad (r = \frac{1}{\xi \omega_n})$

* settling time (t_s) \rightarrow 0 to tolerance band $\Rightarrow \pm 2\% \quad t_s = 4\gamma \quad (r = \frac{1}{\xi \omega_n})$

* time period of oscillation $\rightarrow T_{osc} = \frac{2\pi}{\omega_n}$ sec. $\frac{\pm 0\%}{\text{oscillation before reaching SS.}} \Rightarrow \frac{t_s}{T_{osc}} = \frac{t_s}{2t_p}$

* for exponentially rise time response

$$c(t) = k(1 - e^{-t/\tau})$$

$$t_s = 4\tau \text{ if } \pm 2\% \text{ tolerance}$$

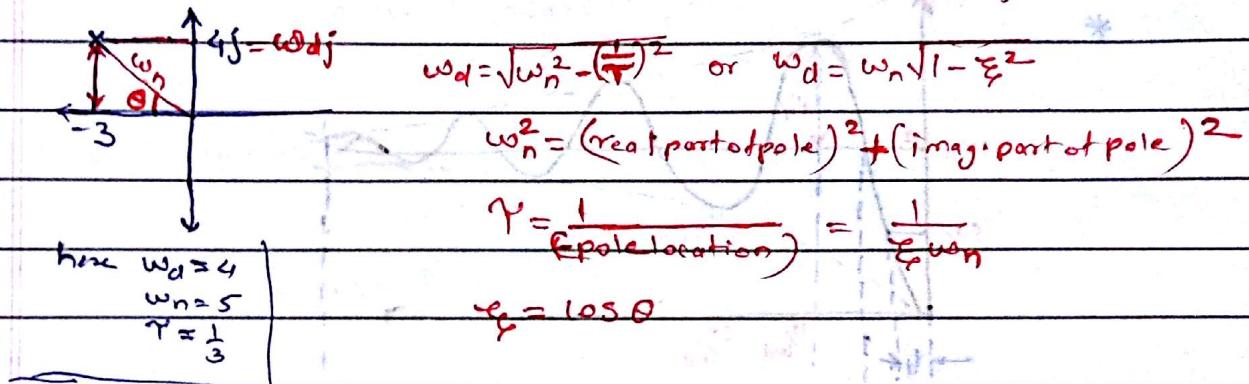
$$0.5 = (1 - e^{-t_d/\tau}) \quad k=1 \because t_d \text{ at } 0.5 \text{ find } t_d$$

$t_r = 2.2\tau$. This formula valid for exponential rise response.

No peaks are exist \therefore No peak time, no peak overshoot.

* given $c(t) = k e^{-3t} \sin 4t$ (impulse response)

here -3 indicates real part 4 indicates imaginary part.



Now we can find time specifications for unit step resp.

using these values of ζ , w_n , w_d , etc.

* $\zeta = 0$ peak overshoot = 100% $\zeta = 1$ peak overshoot = 0%

\therefore as ζ goes from 0 to 1, resp. decreases from 100% to 0%

* Imp points of time domain specifications where ζ is pole structure

① As Real part of pole constant & img. part res. (Pole location) \Rightarrow

\rightarrow Time constant τ remains constant hence settling time is same for all poles.

\rightarrow $w_d \tau \Rightarrow$ hence t_d, t_r, t_p res.

\rightarrow ζ i.e. $\%$ of peak overshoot is T_{res} .

large peak overshoot indicates system is less stable & more oscillatory

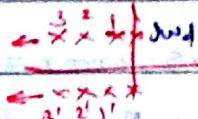
optimum Range of peak overshoot is 5% to 25%

if exceeds 25% system becomes less relatively stable

if below 5% system response becomes very slow. [i.e. 5% to 25%]

* (left and right movement)

② Now if poles moving towards left



$\rightarrow \gamma \downarrow \text{tsses}, t_s \uparrow \text{tsses}$ ω_d constant

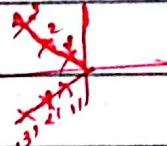
$\rightarrow t_p$ constant, $t_d = 1 + 0.7 \frac{\xi}{\omega_n} \uparrow$, $t_r = \pi \frac{-\cos^{-1} \xi}{\omega_d}$

$\rightarrow \xi \uparrow$ more relatively stable

$\xi \uparrow \text{tsses}, \cos(\theta \downarrow \downarrow) \Rightarrow \gamma \downarrow \text{mp} \uparrow \text{tsses} \therefore$ more relative stable.

③ $\xi \text{ const} = \cos \theta \Rightarrow \text{constant}$

$\gamma \downarrow \text{tsses}$.



$\theta \text{ constant}$ $\xi = \text{const.} \therefore \gamma$ peak overshoot is constant.

$\theta \text{ due to } t_s \uparrow \text{tsses. } t_d, t_r, t_p \uparrow \text{tsses} (\because \omega_d \uparrow \text{tsses})$

④ When $\omega_d \uparrow \text{tsses}$ $t_d, t_r, t_p \uparrow \text{tsses}$ ω_d more

* $\xi \uparrow \text{tsses}$ γ peak overshoot \uparrow tsses.

* $\gamma \uparrow \text{tsses}$ $t_s \uparrow \text{tsses}$.



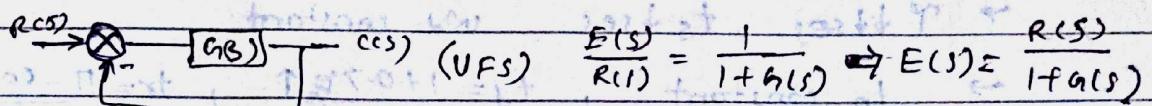
* Poles move upward in jw axis $\omega_d \uparrow \text{tsses}$ ξ constant

* Poles move to left from jw axis horizontally $\gamma \downarrow \text{tsses}$ ω_d constant

(SS E) * Steady state errors :- (deviation of o/p from the i/p)

(S.S. error \Rightarrow the error at $t \rightarrow \infty$)

$$e_{ss} = \lim_{t \rightarrow \infty} e(t) = \lim_{s \rightarrow 0} s E(s)$$



* steady state error depends on (1) type of i/p (2) type of system

* SSE are require to calculate for only closed loop stable system

* SSE are valid only for Unity Feedback System (UFS).

If N.U.F.S. given then convert to UFS & then find SSE

	Step	Ramp	Parabolic
$i/p \propto t^k$	$A u(t)$	$A t u(t)$	$\frac{A t^2}{2} u(t)$
e_{ss}	$\frac{A}{1+k_p}$	$\frac{A}{k_r}$	$\frac{A}{k_a}$
errors const.	k_p -position const.	k_v - velocity error const	k_a - acc. error const
eqns.	$\frac{A}{s^{k+1}} G(s)$	$\frac{A}{s^{k+2}} G(s)$	$\frac{A}{s^{k+3}} G(s)$
$e_{ss} \Rightarrow$	(1) s^0 step	(1) s^1 ramp	(2) s^2 parabolic
Type 1	$\frac{A}{1+k_p}$ (position)	∞	∞
Type 2.	0	$\frac{A}{k_r}$ (velocity)	$\frac{A}{k_a}$ (acceleration)

Type $c = i/p \Rightarrow$ constant / type $> i/p \Rightarrow 0$ / type $< i/p \Rightarrow \infty$

K = calculate from $G(s)$ = coeff. in numerator multiplication
multiplication of denominator coeff.

$A = calculate from (i/p)$ comparing with $(i/p) \rightarrow$ equation on upside

* If given a T.F. verify whether it is OL or CL. \therefore e_{ss} is

valid only for CL & stable system

If OL T.F. given convert to C-L T.F. by $\frac{G}{1+G}$ check stability.

check stability by R.H. criteria or other method to find R.H.S. poles.

If system is CL & stable find e_{ss} or e_{sg} can not find.

Note * Check options before giving ans.

but To find e_{ss} take $G(s)$ only OLTF. to find K (don't do $\frac{G}{1+G}$
& take TF) X don't

* If CLTF, BD or SFG is given,

$$c_{ss} = \lim_{t \rightarrow \infty} [r(t) - c(t)]$$

$$c_{ss} = \lim_{s \rightarrow 0} [R(s) - C(s)] = \lim_{s \rightarrow 0} s R(s) [1 - CLTF]$$

* for unit ramp input $A=1$, unit step input $A=1$

* To find response at $t \rightarrow \infty$ find $y(s)$

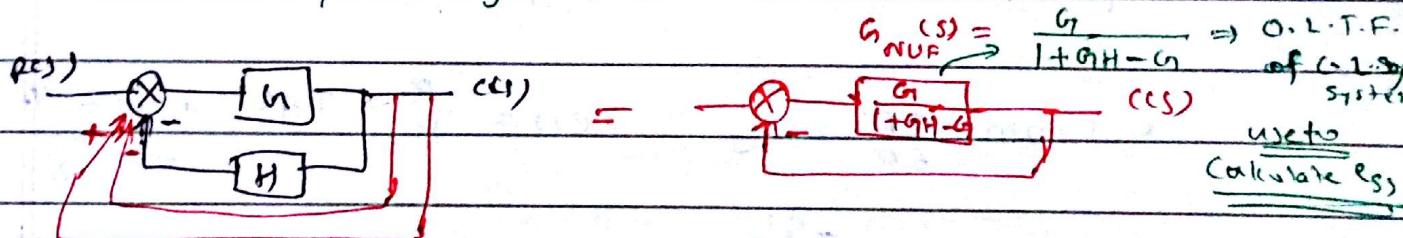
$$(1+t) \lim_{t \rightarrow \infty} y(t) = \lim_{s \rightarrow 0} s Y(s)$$

* steady state o/p (c_{ss}) mean $\lim_{t \rightarrow \infty} c(t) = \lim_{s \rightarrow 0} s [C(s)]$

* The Feed Forward path are used in the system to reduce the effect of noise and disturbance in the system.

* SSE calculated for CL system with Unity Feedback only.

If N.U.F. system is given, convert to U.F. system



* (also to find type & order of NUF system use $G_{NUF} = \frac{G}{1+GH-H}$ & find Type & order)

$$* TF = \frac{K}{1+\tau s} e^{-Ds} \quad K = \text{static gain}, \tau = \text{time constant}, D = \text{transport delay}$$

* If p is sinusoidal o/p is also sinusoidal with extra magnitude & phase angle ^{out}m obtained from T.F. & static gain also put $s=j\omega$ in T.F. & get ev value from ip now calculate mag. & angle

* If K is spring constant M is mass of instrument then

$$T_p = 2\sqrt{Km} \text{ to prevent overshoot in step response}$$

* Type increases S.S.E. & vice versa

Introducing integral in F.W. path \Rightarrow Type of system \Rightarrow ess vs. type
i.e. diff. T.F. i.e. F.P. Gain vs. type here \Rightarrow ess.

* JSE (integral of squared error) = $\int_0^\infty e(t)^2 dt$ $e(t) = \text{ILT of } E(s)$

$$\frac{E(s)}{R(s)} = \frac{1}{1 + GH} \quad (\text{H = 1})$$

: unity feedback

(Response at $t \rightarrow \infty$ means final value)

* T.F. of Thermocouple: $\frac{C(s)}{R(s)} = \frac{K}{1 + TS}$ $E(s) = R + u(s)$
update

* As system type \Rightarrow S.S.E. is eliminated

* With Feedback $\gamma \rightarrow \infty$ & hence response decay quickly.

* Transient response depends on system only

S.S. response depends on input only.

* presence of Non-linearities in system introduces S.S.E.

$17\text{ rpm} = \frac{\pi}{60} \text{ rad/s}$ $E(s) = \frac{R}{Kv}$ Type 1 order 2 $\Rightarrow p = \text{const}$
 $R = \frac{\pi}{3}$

* for $m_p = e^{-\frac{\pi s}{\sqrt{1-\zeta^2}}}$ $\Rightarrow IR \rightarrow (t) = -4e^{-t} + 6e^{-2t}$
Not in% $\text{for } t \geq 0 \text{ U.S.R. } (t) = \int_0^t -4e^{-t} + 6e^{-2t}$
solve

* $\frac{E(s)}{X(s)} = \frac{1}{1 + GH}$ $\lim_{s \rightarrow 0} S.E(s)$
 $ess = \lim_{s \rightarrow 0} S.E(s)$

* stability.

* LTI system is said to be stable if

① Bounded i/p gives Bounded o/p.

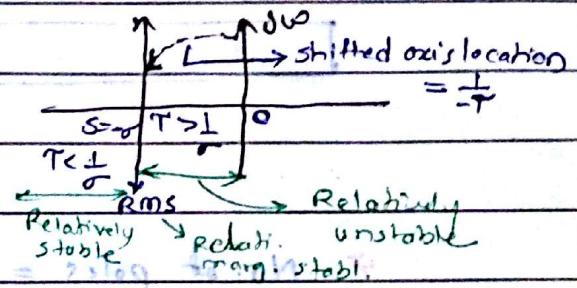
② If i/p is '0' o/p must be 0 irrespective of initial conditions.

absolutely stable :- stable for all values of system parameters.

conditional stable :- stable for certain range of system parameters.

Marginal, critical, limitally stable :- Bounded i/p & o/p maintains constant amplitude and frequency. non-repeated poles on j-axis.

* Relative stability :- applicable for stable systems only using relative stability concept we can find T , t_s , & t_{ss} .



* For finding CL sim. stability by using RH criteria we need characteristic equation $(1+G(s))H(s)$

whereas to find stability by techniques like RL, BP, NP we require a OLTF at Unity or non-unity feedback sim.

* R-H (Routh-Hurwitz) Criterion.

→ main purpose is to find ^{number of} poles on Right hand side.

→ To find closed loop system stability.

→ To find no. of CL poles lies on left, right or imaginary axis of s-plane

→ To find range of sK -values of CL sim. stability.

→ To find k -values for system to become marginal stable or undamped

→ To find Natural frequency or undamped oscillations.

→ To find relative stability & hence from relative stability

constants we can find T , t_s , t_{ss}

characteristic eqn. $a_n s^n + a_{n-1} s^{n-1} + a_{n-2} s^{n-2} + \dots + a_1 s + a_0 = 0$

s^n	a_0	a_2	$a_4 a_2 - a_0 a_1$	$+/-$	$s^n > 0$	$+$
s^{n-1}	a_1	a_3	a_5	$+/-$	stable	$-$
s^{n-2}	$(a_0 a_2 - a_0 a_3)$	$(a_0 a_4 - a_0 a_5)$		$+/-$		$+$
s^0	a_0					$+$

Condⁿ for s/m stability in R-H criterion: II

- ① all coeff in 1st column have some sign no coeff in 1st column 0
- ② if any sign change occurs in 1st column s/m unstable.

no. of signchange in 1st column = no. of poles on RHS of S-plane.

* $as^2 + bs + c = 0$ $a, b, c > 0$ or $< 0 \Rightarrow$ s/m stable. : some sign

$b=0, ac < 0$ or $ac > 0 \Rightarrow$ marginal stabl.

* $as^3 + bs^2 + cs + d = 0$ $bc > ad \Rightarrow$ s/m stable

$bc = ad \Rightarrow$ marginal stabl. \downarrow take even powers

$bc < ad \Rightarrow$ unstable \Rightarrow $bs^2 + d = 0$ \Rightarrow $s = \pm j\sqrt{\frac{d}{b}}$

freqⁿ of oscillatⁿ \Rightarrow

$$\omega_n = \sqrt{\frac{d}{b}}$$

* No. of poles = highest power of char. eqn.

No. of poles in RHS = no. of sign changes.

* Case 1 if any elem in a 1st column is 0 replace it by positive constant ξ & continue process. Finally substitute $\xi = 0$ & check the sign changes.

* Case 2 if all the elements in a row becoming 0. find AE

(Auxillary eqn) of above row take diff. of AE. wrt. s.

then take coeffs of s^k & put on a row & continue the process

→ The row of zero occurs when poles are symmetrical

about origin, i.e., AE should consist only even power of s terms

∴ roots are symm. about origin. To prove back at

→ The AE gives no. of poles on jw axis. If 0 is marginally stable

→ in R-H tabular form if only one Row of zero occurs

all the coefficients in the 1st column is +ve then the s/m

is marginally stable. ∵ poles on jw axis & non repeated.

→ if sign changes occurs below AE there should be symmetrical

pole in RHS & LHS of the s-plane.

if sign change in the above of AE no. symmetrical pattern

the left to the pole placed in the right side.

* No. of times repeated zero row occurs in R.H. Table given no. of times repeated poles on jw axis, sym. about origin.
if only one time zero row Non-repeated poles regular.
sym. poles about origin.

* if only one zero row occurs doesn't mean poles on jw axis.

* for conditional stable system

Find K value from eqn of K in 1st column & comparing with 0
(expns of K) > 0 stable, < 0 unstable, $= 0$ margin. stable ~~marginally stable~~
~~for $\sigma = 0$~~

* To find relative stability first find whether given system is stable or not. ^{relative} stability only given for stable system.

stability about $s = -\omega$ replace s by $(z - \omega)$ and find R.H table

conclude the relative stability or stability in

function of slope position

* Limitations of RH criteria: will be work on

① By using RH criteria we can't identify the nature of the system bcz we can't determine the exact locat'n of the pole

② RH criteria not applicable for exponential or sign cosine terms. bcz it gives infinite series. e.g.: RH criteria applicable for only finite no. of ~~prob~~ terms available

But using RH criteria we can get approximate sol'n to the exponential term. by neglecting higher order terms i.e. $e^{-st} \approx 1 - st$

* Root locus.

- To find CL sm stability.
- To find range of K -values for sm stability.
- to find K -values for sm to be marginal stable.
- find natural freqn of oscillation when sm is marginal stable.
- K values to become sime undamped, underdamped, (D, OD)
- To find relative stability. RL moving left more relatively stable else.
RL branch moving RHS side, less relatively stable.
- Best method to find absolute stability.

* Root locus means close loop poles path by varying K value from 0 to ∞ .

* We can't draw root locus by characteristic eqⁿ bcz as order increases finding roots is difficult.

∴ To draw R.L. we use O.L.T.F. But for the stability analysis

we used C-L.T.F. but for drawing R.L. going to plot s-plane with external traces with minimum trace and

* Relation Betw OLT & CLTF - poles & zeros

→ closed loop poles are nothing but open loop poles & open loop zeros multiplied by const factor of system gain K . $[O(s) + K N(s)] = 0$

① $K=0$ closed loop poles = open loop poles.

② $K=\infty$ closed loop poles = open loop zeros.

$\Rightarrow K \uparrow_{s \rightarrow 0} [0 \rightarrow \infty] \Rightarrow O.L. \text{ poles} \rightarrow O.L. \text{ zeros}$

$K \uparrow_{s \rightarrow \infty} [\infty \rightarrow 0] \Rightarrow O.L. \text{ zeros} \rightarrow O.L. \text{ poles}$

* To draw R.L. no. of poles = no. of zeros.

if zeros are less consider zeros are ∞ .

R.L. starts at poles and ends at zeros.

* Angle condition: (A.C.)

-ve feed back chara. Eqⁿ = $1 + G(s)H(s) = 0$

$$\angle G(s)H(s) = \text{odd multiples of } \pm 180^\circ = \pm [2q+1]180^\circ$$

Direct R.L. 180° Rules.

$q = 0, 1, 2, \dots$

+ve feedback chara. Eqⁿ = $(1 - G(s)H(s)) = 0$

$$\angle G(s)H(s) = \text{even multiples of } +180^\circ = (2q)180^\circ \quad q = 0, 1, 2, \dots$$

Inverse R.L. complementary RL. 0° Rule

* Purpose of Angle condition is to check if any point lies on R.L.

or not, i.e. All pts on R.L. must satisfy angle condition.

Put (s) value in $G(s)H(s)$ at given pt & find \angle if 180° Rule ✓

if 0° Rule ✗

* Magnitude condition: (m.c.) $|G(s)H(s)| = 1$ OR \angle to align w.r.t. 0° Rule

$$|G(s)H(s)| = 1 \quad \text{at any pt on R.L.}$$

purpose is to find sign gain at any pt.

if pt is given find if pt is on RL or not by 2 conditions

then by mag. condn $|G(s)H(s)| = 1$ find value K at that point.

done with help of table derived J.P.

following points are imp for the construction of R.L.

① Symmetrical :- RL is symmetrical about Real axis.

NP is no. of poles if plotted on semilog paper which is non linear

NP is symmetric about the midline

② No. of root locus branches or no. of loci :-

No. of R.L. branch is max no. of (Poles or zeros)

③ R.L. branches :- to avoid extra loops

A point to exist on R.L. the sum of the poles & zeros on RHS.

of the pt. should be odd even \Rightarrow to zero

Never apply of A.C. & m.c. for poles & zeros.

(4) **Asymptotes** :- Asymptotes are R.L. branches approaching to ∞

No. of Asymptotes $n = P - Z$

angle of asymptotes $\Theta = \frac{(2q+1)180^\circ}{P-Z}$ $q = 0, 1, 2, \dots, P-Z-1$

Asymptotes are symmetrical about Real axis.

(5) **Centroid** : Intersection of asymptotes on Real axis

$$\text{Centroid} = \frac{\text{sum of real parts of poles} - \text{sum of real parts of zeros}}{P - Z}$$

→ centroid may or may not be on R.L. branch

- from centroid we draw asymptotes with an angle of asymptotes.

(6) **Break point** : pt at which two or more poles meets

Root Locus branches enters or leaves the real axis with

an angle of $\pm 180^\circ$ when n is no. of poles at the break point

BAP → Break away point :- RL leaves Real axis

BIP → Break in point :- RL enters in Real axis.

→ minimum 1 break away pt b/w two adjacent pole if RL branch is exist

→ minimum 1 break in pt b/w two adjacent zero if in between then

RL branch exist in between the zeros.

→ left most side of left most zero if exist RL branch then at least

one break in point would be there

→ If there exist RL branch of left most side of left most pole.

then minimum one breakaway point.

(7) To find Break point

① characteristic eqⁿ $\rightarrow H(s)H(s)$ divided by s^n to get

② Rearrange eqⁿ in the form of $K = f(s)$

③ diff. w.r.t. s & make it equal to 0

④ roots of $\frac{dk}{ds} = 0$ gives Break point

⑤ take valid break point according to R.L. branch.

* Denominator of OTF diff. w.r.t. s & equat to 0 for values of s

⑦ Intersection point with imaginary axis.

- ① Form characteristic eqn
- ② Write R.H criteria tabular form.
- ③ find k-marginal value.
- ④ form the AE equation
- ⑤ Root of AE equation gives valid and invalid intersection pt.

For valid intersection pt. with imag. axis, its marginal value is +ve

⑧

Angle of departure and arrival

Angle of departure is calculated at complex conjugate poles

Angle of arrival (departure) is calculated at complex conjugated zeros

- * Angle of departure (ϕ_d) \rightarrow it gives the angle which poles leaves from initial position i.e. R.L. branch starts at an angle of ϕ_d (at complex pole)

$$\phi_d = 180^\circ + \angle GH = 180 - \phi \quad [\text{where } \phi = \angle \phi_p - \angle \phi_z]$$

- * Angle of arrival (ϕ_a) \rightarrow it gives \angle at which poles terminated at complex zeros. i.e R.L. branch ends at an angle ϕ_a (at complex zero)

$$\phi_a = 180^\circ - \angle GH = 180 + \phi \quad [\text{where } \phi = \angle \phi_p - \angle \phi_z]$$

(while finding angle put $s = \text{value of pole at which } \angle \text{ is finding}$)

* Method to draw Root locus:

1. Identify the real axis root locus branches & break points

2. find the angle asymptotes and centroid (if required)

Find angle of departure & arrival.

3. Vary k values from 0 to ∞ & identify the path from pole to zero. The R.L. is symmetrical about real axis, & all the poles must reaches to zero.

* find nature of system OD, CD, UD, undamped etc. to get k value

for diff. nature we require k value at Break point then conclude

nature of system w.r.t. k value

* at break point $s \in ic\text{C}_0$, to find k at C_0 use magnitude condⁿ.

* Step by step procedure

- ① Locate poles & zeros in S-plane
- ② Identify no. of branches & starting point (poles) & ending points of branches.
- ③ Find centroid & asymptotes.
- ④ Break points (if necessary)
- ⑤ intersection point on jw axis.
- ⑥ angle of departure if complex poles or arrival if complex zero
- ⑦ conclude for range of K values & nature of systems.

~~extra steps (not relevant to bisection) to calculate the signs
of coefficients. To find poles & zeros use CLTF on the left
and most * to find Break point & to find intersection pt on jw axis
(in a karnaugh map) used characteristic eqn of close loop T.F.
to find angle of departure or arrival use T.F. G(s)H(s) CLTF
however when no. poles = no. of zeros centroid & asymptotes not required.~~

~~* When 1B.P. = centroid and all poles are symmetric about B.P.~~

CLTF for this case is $\frac{1}{s + p}$ (see example)

~~divide along the real axis (centroid) slope pointing right~~

~~if 1B.P. δ = Real part of complex pole~~

~~string formed & reflected w.r.t. from zero locus will pick up if complex poles are very near to Real axis~~

~~if Symmetric 1B.P. \Rightarrow 1B.P. + 2B.P. complex poles are away from real axis~~

~~→ z-axis form (midpoint of 1B.P. + 2B.P.) or 1B.P. + 1 pair of complex B.P.~~

* Whenever complex poles are very close to real axis as compared to imaginary axis then no. of B.P. on real axis increases.

* spirule is used to draw root locus & calibrate it in terms of variable parameter.

* Bode plot:

- To draw frequency response of OLT.
- To find close loop system stability
- To find gain margin phase margin, gain crossover frequency & phase crossover frequency.
- To find relative stability by using gain margin and phase margin.

gain margin large: more relatively stable but S_m is low.
 if $g_m \& p_m$ is less then less relatively stable. But S_m becomes oscillatory.
 $\therefore g_m \& p_m$ should be optimum. $g_m \Rightarrow 5 \text{ to } 10 \text{ dB}$ $p_m \Rightarrow 30^\circ \text{ to } 40^\circ$

Bodeplot = ① magnitude plot + ② phase plot.

magnitude plot: → $\frac{\text{Decade}}{\text{(times change in frequency)}}$ $w_2 = 10w_1$, decade means $\frac{20 \text{ dB}}{\text{decade}}$

* Procedure to draw the bodeplot.

- ① S is replaced by $j\omega$ to convert it to frequency domain.
- ② write magnitude convert it into dB, mag. in dB is $20 \log |G(j\omega)H(j\omega)|$
- ③ find the phase angle by using $\tan^{-1} \left[\frac{\text{imaginary part}}{\text{real part}} \right]$
- ④ vary ' ω ' from minimum to max. value. & draw mag. & phase plot.

* shift in the magnitude plot depends on the K value. Whereas the phase plot is independent of K value.

- * n poles at origin \Rightarrow mag. plot starts with mag. +20n with slope of $-20n \text{ dB/decade}$.
- * n zeros at origin \Rightarrow mag. plot starts with mag. -20n with slope of $+20n \text{ dB/decade}$.
- * $\frac{K}{s^n}$ same as n poles at origin but graph is shifted by $20 \log K$ upward. If $K=1$ mag. plot pass through $\omega=1$ axis of magnitude plot is 0dB line.

* n-infinite poles $\Rightarrow H(s) \propto \frac{1}{(s\tau_1 + 1)^n}$ which result at

$$\text{mag. in dB} = -20n \log_{10}(\omega\tau_1)^2 + 20n \text{ const.}$$

$$\angle \phi_{\text{actual}} = n \tan^{-1}(\omega\tau_1)$$

* n-infinite zeros $\Rightarrow H(s) \propto (s\tau_1 + 1)^n$

$$\text{mag. in dB} = 20n \log_{10}(\omega\tau_1)^2 + 20n \text{ const.}$$

$$\angle \phi_{\text{actual}} = n \tan^{-1}(\omega\tau_1)$$

* corner freq. $\omega_c = \frac{1}{\tau_1}$ freq. at which slope is changing.

< corner freq. slope of pole & zero is $+20n \text{ dB/dec}$ & $-20n \text{ dB/dec}$.

> corner freq. slope of pole is $+20n \text{ dB/dec}$ & $-20n \text{ dB/dec}$.

slope of zero is $+20n \text{ dB/dec}$

(n is repeated no. of poles)

$\angle \phi$ for poles $= -90^\circ n$ & $\angle \phi$ for zeros $+90^\circ n$.

* corner freq. $\omega_c = \sqrt{\frac{(T_F)}{(P)}} = \sqrt{\frac{A_{\text{asymp}}}{A_{\text{actual}}}}$

→ The error at corner freq. is very large. either above or below corner frequency the error decreases symmetrically below one decade whatever the error exist after 1 decade. the error is same. error is symm. about corner freq.

* When f^h is given convert it into standard Transfer fn

$$\frac{K(1+sT_a)(1+sT_b)}{(1+sT_1)(1+sT_2)}$$

& inverse of T gives corner frequencies.

* Eg.

* change in slope is slope of no. of finite poles/finite zeros at particular corner frequency. ~~\propto by $\pm 20d$~~ -20 for pole $\frac{-20}{+20}$ for zero
 $\omega_0 = k \left(1 + \frac{S}{Z}\right)^m$ change in slope at $\omega = 2$ is $\pm 20d$ dB/dec
 $\left(1 + \frac{S}{Z}\right)^n$ change in slope at $\omega = 3$ is $+20m$ dB/dec
 $\therefore 20d = \pm 10m$

* Slope of "line betw" the two corner frequency from ω_1 to ω_2 . consider T.F. up to ω_1 only, get diff. betw poles & zeros. (d)
 the slope = $\pm 20d$ ($+if Z > P, -if P > Z$)
 $29.1A\phi + 29m\phi = 29m\phi$

* Slope around corner frequency. :- around ω_c means
 if ω_c be betw ω_1 & ω_2 ($\omega_1 < \omega_c < \omega_2$ & above freq. terms) in T.F. get slope
 w.r.t ω above ω_1 , ω_2 included only terms that is $(\omega_1 < \omega_c < \omega_2)$
 w.r.t ω below ω_1

* To find T.F. from magnitude plot.

- ① observe the initial slope it gives the poles & zeros at origin
- ② find change in slope at each corner frequency if change is +ve consider finite zeros ($\frac{\text{slope}}{20}$) no. of zeros. if slope -ve then poles.
- ③ find K value. by using known magnitude at known frequency
 To find K value take initial slope. if we know mag.

Rebind for some corner freq. take transfer f.c. up to that corner frequency and then take log eqⁿ & get K value.

$$\begin{array}{c} \text{+20dB/dec} \\ \text{---} \\ \omega_1 \end{array} \Rightarrow K = (\omega_1)^n \quad \begin{array}{c} \omega > \omega_1 \\ \text{---} \\ \text{+20dB/dec} \end{array} \Rightarrow K = (\omega_1)^{-n}$$

$$\begin{array}{c} \text{+20dB/dec} \\ \text{---} \\ \omega_1 \end{array} \quad m = 20 \log K + 20 \log \frac{\omega}{\omega_1} \quad \begin{array}{c} \text{+20dB/dec} \\ \text{---} \\ \omega_1 \end{array} \quad m = 20 \log K + 20 \log \omega,$$

$$* \text{ Slope} = \frac{dm}{d \log \omega} = \frac{m_2 - m_1}{\log \omega_2 - \log \omega_1}$$

* log base in these calculation is 10 * dont take ln.

* minimum phase system: s/m in which all the finite poles and finite zero's lies in the left of s-plane.

it gives phase angle less than or equal to -90°

* All pass system: all poles lies in left & all zeros are lies in right.
 $m=1 \quad \phi = \pm 180^\circ$.

* Non-minimum phase system: one or more poles or zeros or both lies on right. this system gives more negative angle at $\omega=\infty$.

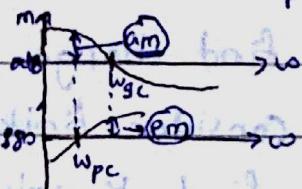
$$\phi_{NMPs} = \phi_{MPS} + \phi_{ALPS}.$$

* Gain cross over frequency: ω_{gc} is a freq at which magnitude $|G(s)H(s)|=1$ or 0 dB. the phase value at ω_{gc} is called phase margin. (PM)

* Phase cross over frequency: ω_{pc} is a freq. at which change in phase \angle is -180° . the gain value at ω_{pc} is called gain margin (GM)

$$GM_{dB} = -20 \log |G(j\omega)H(j\omega)| \Big|_{\omega=\omega_{pc}} \text{ dB}$$

$$PM = 180^\circ + \angle G(j\omega) \Big|_{\omega=\omega_{gc}}$$



When $\omega_{pc} > \omega_{gc}$	$GM > 1 \& PM +ve \Rightarrow$ stable
$\omega_{pc} = \omega_{gc}$	$GM = 1 \& PM = 0^\circ \Rightarrow$ marginal stable
$\omega_{pc} < \omega_{gc}$	$GM < 1 \& PM = -ve \Rightarrow$ unstable

$$* G(s)H(s) = \frac{1}{s} \Rightarrow \omega_{pc} = \infty \quad (\text{constant phase of } -90^\circ) \Rightarrow \text{stable}$$

$\Leftrightarrow \omega_{pc} > \omega_{gc}$

$$= \frac{1}{s^2} \Rightarrow \omega_{pc} = \omega_{gc} \quad (\text{constant phase of } -180^\circ) \Rightarrow \text{marg. stable}$$

$\Leftrightarrow \omega_{pc} = \omega_{gc}$

$$\text{Actually } \omega_{pc} \text{ doesn't exist.} \quad = \frac{1}{s^3} \Rightarrow \omega_{pc} = 0 \quad (\text{constant phase of } -270^\circ) \Rightarrow \text{unstable}$$

$\Leftrightarrow \omega_{pc} < \omega_{gc}$

* Making complex bode plots.

* n -complex poles $\Rightarrow G(s) = \left(\frac{w_n^2}{s^2 + 2\xi w_n s + w_n^2} \right)^n$

at $s=j\omega$ & flat magnitude, we get $M = \left[\frac{1}{\sqrt{\left[1 - \left(\frac{\omega}{\omega_n} \right)^2 \right]^2 + \left[2\xi \left(\frac{\omega}{\omega_n} \right) \right]^2}} \right]^n$

in dB Mag

$$dB(A_{Actual}) = -20n \log \left[\sqrt{\left[1 - \left(\frac{\omega}{\omega_n} \right)^2 \right]^2 + \left[2\xi \left(\frac{\omega}{\omega_n} \right) \right]^2} \right]$$

(ω_n = corner freq)

$$\phi_{Actual} = -n \tan^{-1} \left[\frac{2\xi \left(\frac{\omega}{\omega_n} \right)}{1 - \left(\frac{\omega}{\omega_n} \right)^2} \right]$$

M correction at C. Freq. = $-20n \log 2\xi$

* n complex poles zeros. mag same as above & ϕ too only change in sign. \pm

zero M is M^{-1} poles

ϕ correction at cor. freq = -90°

M correction & ϕ correction also ^{in sign.!} only change - to +

* Correction at corner frequency depends on ξ

other than corner frequency depends on ξ and ω in mag.

Whereas in phase plot correction at cor. freq. is constant

other than corner freq. depends on ξ & ω .

$$* \left(1 + \frac{s}{\omega_n} + \frac{s^2}{\omega_n^2} \right) \text{ compare with } \left(\frac{s^2 + 2\xi w_n s + w_n^2}{w_n^2} \right)$$

$$= \left[1 + \left(\frac{2\xi}{\omega_n} \right) s + \left(\frac{s}{\omega_n} \right)^2 \right]$$

* B.W. is inversely proportional to noise but increases noise assymp.

from ω_n to $\omega_n / \sqrt{2}$ \Rightarrow $B.W. \propto \frac{1}{\sqrt{2}}$

from ω_n to $\omega_n / \sqrt{3}$ \Rightarrow $B.W. \propto \frac{1}{\sqrt{3}}$

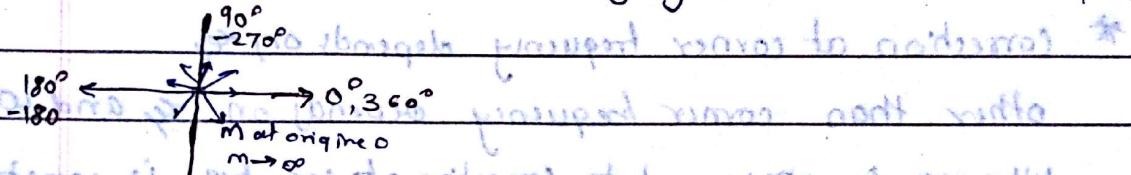
* Polar plots

- To draw frequency response of CLTF.
- To find CLSM stability.
- To find gain margin and phase margin.
- To find relative stability.
- The polar plot is not a complete freq. plot. \because frequency range $0 \rightarrow \infty$
Nyquist plot is complete freqn. plot in $-\infty \text{ to } +\infty$.
- Polar plot is nothing but magnitude vs phase plot.

To draw polar plot

Std procedure take various values of $M \& \phi$ of $G(j\omega)H(j\omega)$ by putting

Various values of ω ranging from $0 \rightarrow \infty$ & plot graph.



$$\text{Ans} \Rightarrow \text{one change } G(j\omega)H(j\omega) = \frac{K(s+2)}{(s+1)^2} \text{ etc put } s=j\omega .$$

$$\angle K = 0^\circ$$

$$(j\omega + 2)(s+2) \quad \angle G(j\omega)H(j\omega) = \angle K + \angle j\omega \cdot \frac{\omega}{2} - \angle \tan^{-1}(\frac{\omega}{1})$$

$$[j\left(\frac{\omega}{1}\right) + 2\left(\frac{\omega^2}{1}\right) + 1] = |G(j\omega)H(j\omega)| = \sqrt{(j\omega)^2 + 2^2}$$

* if mag. at $\omega=0 \geq$ mag. at $\omega=\infty$ then go for following
procedure. method to draw polarplot

① Find the mag. & phase. at $\omega=0$ i.e. $m_1, \angle \phi_1$

② Find mag & phase. at $\omega=\infty$ i.e. $m_2, \angle \phi_2$

③ ending direction $\rightarrow \phi_1 - \phi_2 \Rightarrow$ +ve \rightarrow clockwise.

-ve \rightarrow Anticlockwise.

④ Starting direction considered to the transfer f/n. It should have only +ve signs. terms.

If finite pole is near to imag. then direction is CW \rightarrow zero

If finite zero is near to the imag. the direction is ACW \rightarrow ACW.

If zeros at origin \rightarrow std procedure \quad If $P > Z$ \rightarrow another procedure

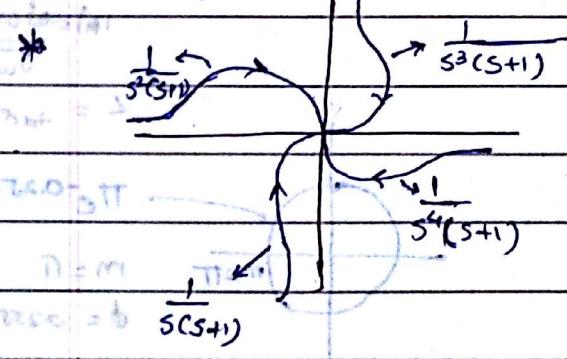
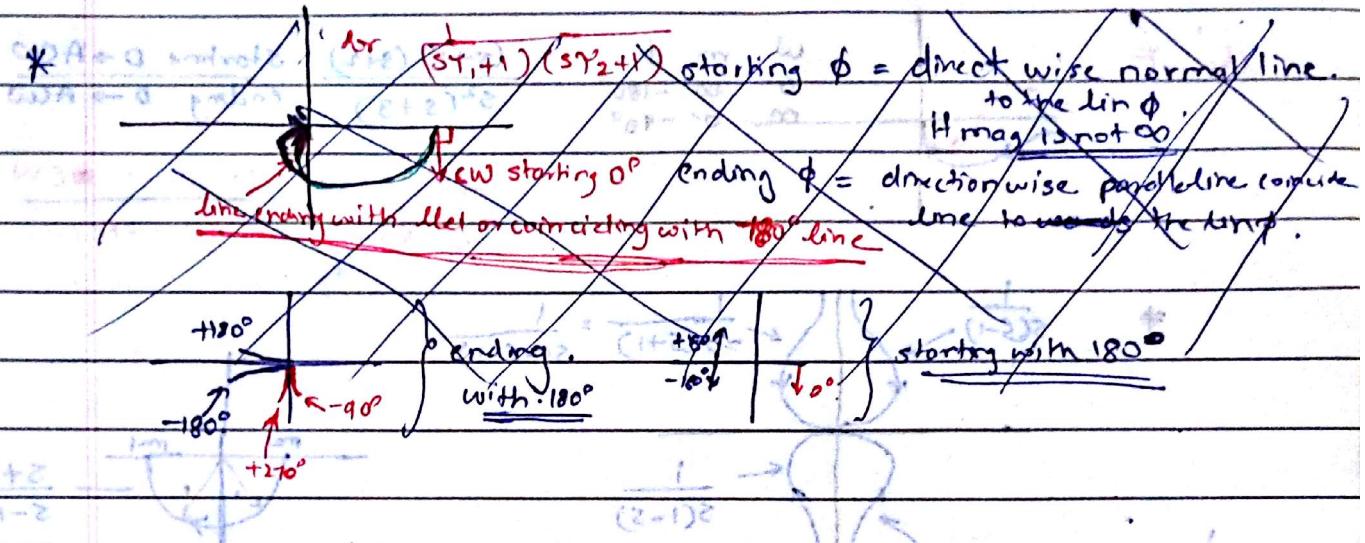
$$G(s) = \frac{K}{(s+1)(s+2)(s+3)}$$

(addition of each finite pole shift
ending angle by -90°)

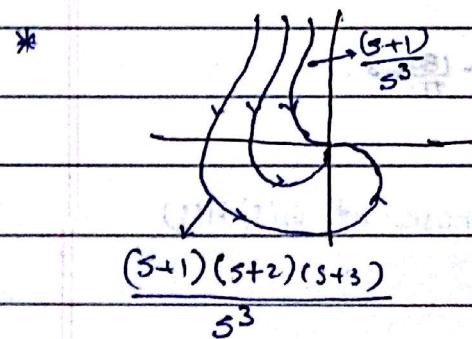
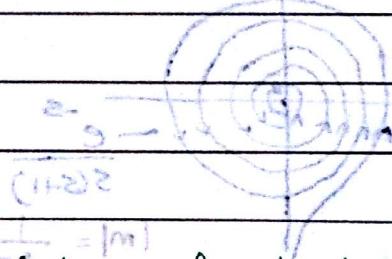
$$\omega = \sqrt{\frac{T_1+T_2+T_3}{T_1 T_2 T_3}}$$



$$\omega = \sqrt{T_1 T_2 + T_2 T_3 + T_1 T_3}$$



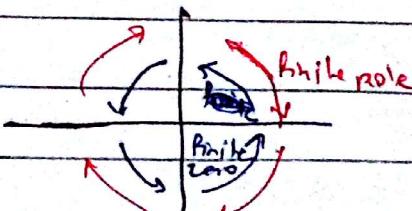
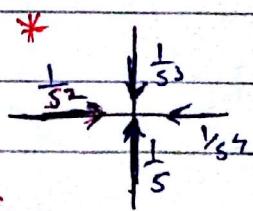
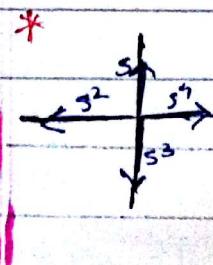
(addition each pole cut origine shiftte.
total plot by -90° in cw direction)



(addition of each finite zero shift

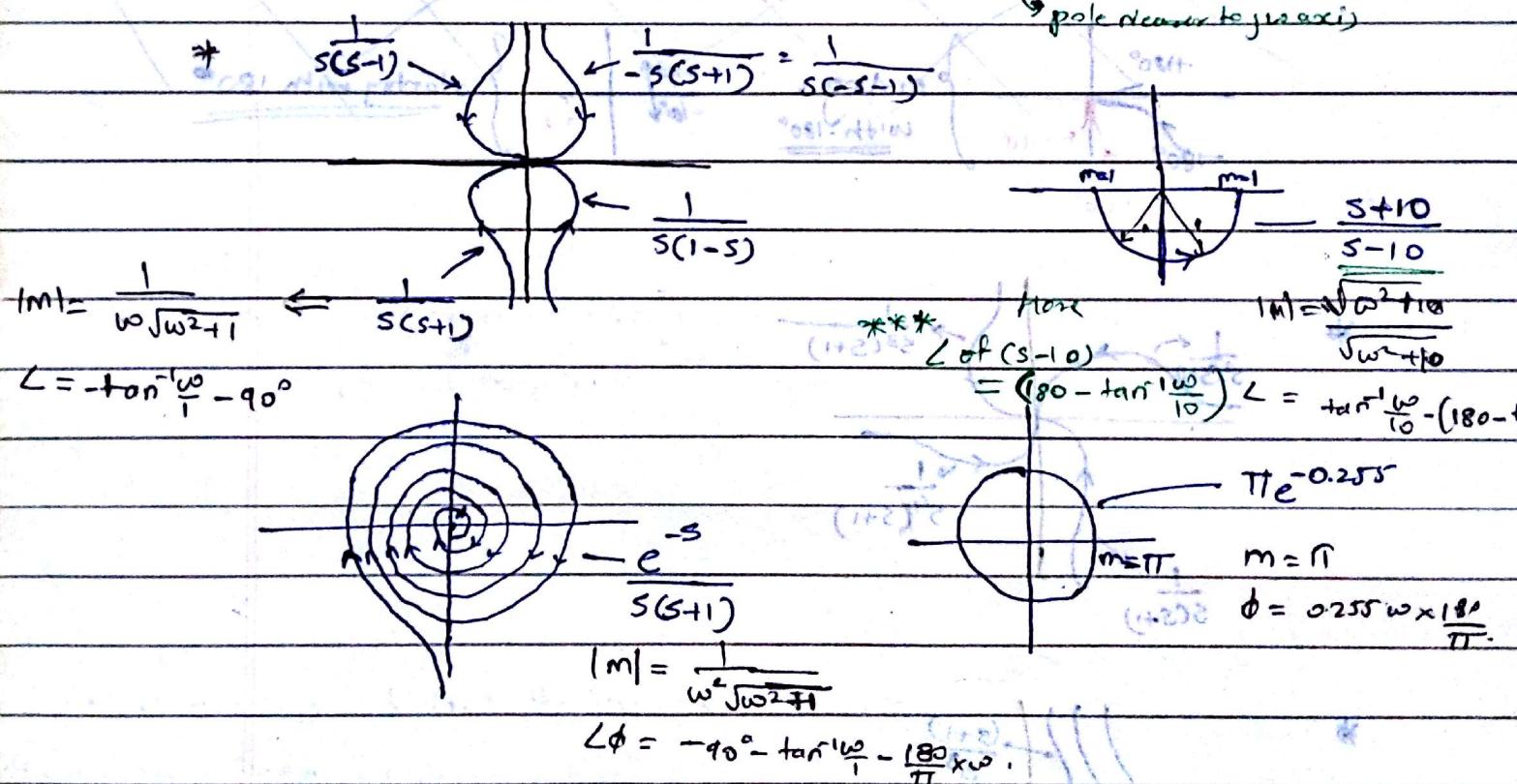
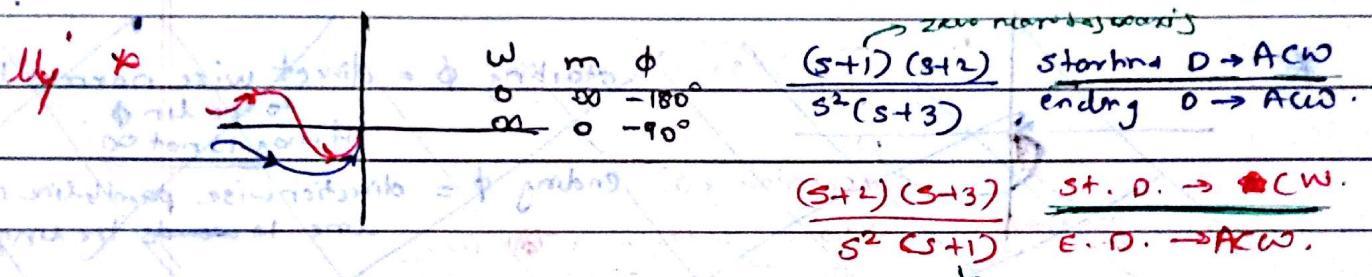
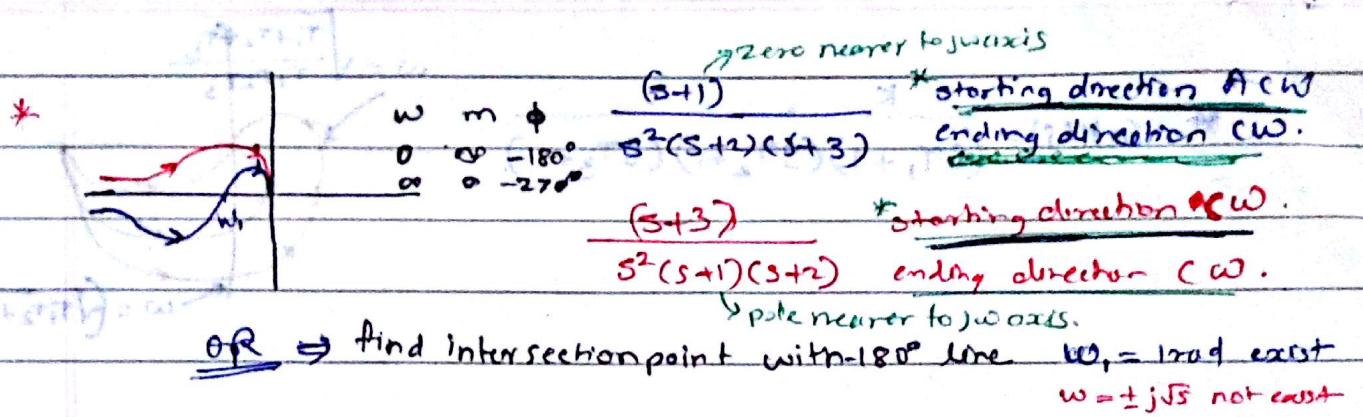
the ending angle by $+90^\circ$ in A clockwise

* Whenever transfer $T(s)$ consist only poles or only zeros at origine.
polar plot is angle line



$$MTF = \frac{1}{s^2} \dots \text{plot}$$

Starts from -180° line
& so on..



* Polar plot is a magnitude & phase of $G(s)H(s)$

$$(s+2)(s+2)(s+2)$$

* Nyquist plot.

- To draw freq. response of open loop transfer function
- To find number of closed loop poles in the right of s-plane.
- To find the range of k-value for system stability.
- To find G_m , P_m , ω_{gc} & ω_{pc} .
- To find relative stability using G_m & P_m .

Nyquist stability criteria developed by using principle of arguments.

Principal of arguments → It states that if there are poles and zeros are enclosed by the randomly selected closed path then the corresponding $(G(s)H(s))$ plane encircles origin with $P-Z$ times i.e., $n = P-Z$.

$$* N = P - Z = +2 \quad \text{ACW direction}$$

$$* N = P - Z = -2 \quad \text{CW direction}$$

$$* N = P - Z = 0$$

* the principle of argument concept is applied to the total right of the s-plane with a radius called Nyquist count.

* Nyquist stability analysis is right of s-plane analysis.

$$* N = P - Z \quad \text{here } P \rightarrow \text{Poles of CL in the RHS of s-plane.}$$

$$* \left[1 + G(s)H(s) \right] \quad \begin{aligned} &\text{i.e. OLTF poles in the RHS of s-plane} \\ &\text{characteristic equation} \end{aligned}$$

$$Z \rightarrow \text{zeros of CL in the RHS of s-plane}$$

$$\quad \quad \quad \text{i.e. CLTF poles in the RHS of s-plane.}$$

$$N \Rightarrow \text{No. of encirclement about critical point}$$

$$(\text{critical pt} \Rightarrow (-1 + j0))$$

To become C/L stable there shouldn't be any CL pole in the RHS plane CL pole is nothing but zeros of CE.

$$\therefore Z = 0, \quad N = P$$

* ∴ No. of encirclement about the critical point must be equal to poles of CE which are nothing but O.L.T.F. poles

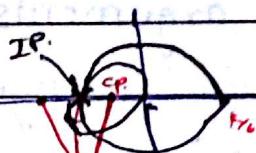
$$N = P \quad \& \quad Z = 0$$

while drawing N.P.

- ① draw polar plot of given T.F. $H(s)H(s)$, then take mirror image with opposite director orientation must be continuous to form a loop.
- ② if in T.F. open poles at origin, then encirclement of $180^\circ \times n$ times with ∞ radius. & direction of this must be continuous with above polar plot. see example in book note do clear.

$$+ K \\ (s+1)(s+2)(s+3)$$

\Rightarrow N.P. \Rightarrow



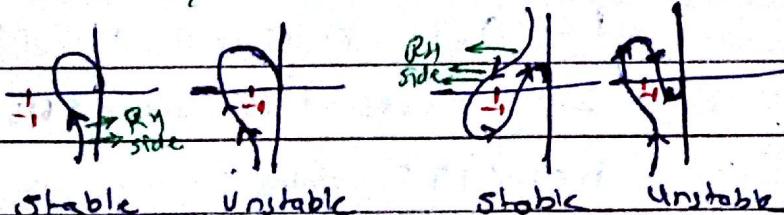
3 cases for c.p. according to IP

→ find I.P. at -180° line like in polar plots.

→ for stability $N=P$. (here $P=0$. $\therefore N$ should be zero for stability.
∴ IP should $<$ CP. for stability. find IP & conclude for Gurey values
(at IP = CP marginal stable))?

* In any problem given polar diagram means there is no effect of infinite radius half circle on system stability.

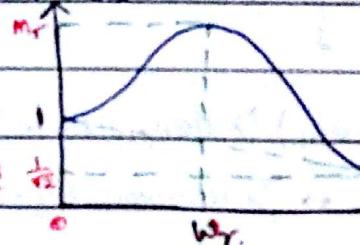
* whenever the polar plot intersects -180° line with magnitude less than 1, the s/m is stable. $w_p > w_g$.
 $(\Sigma P = C.P.)$
OR if equal to 1 then s/m is margin. stable : $w_p = w_g$
OR if greater than 1 the s/m is unstable. $w_p < w_g$.



* Frequency domain Specifications

$$\underline{V(j\omega)} = \frac{N(j\omega)}{D(j\omega)} = \frac{1}{s^2 + 2\xi s + \omega_n^2}$$

$$\omega_n = \sqrt{LC}, \xi = \frac{R}{2\sqrt{LC}}, \omega_r = \frac{1}{2\xi}$$



* Resonant freqⁿ (ω_r) = $\omega_n \sqrt{1-2\xi^2}$ rad/sec
Freqⁿ at which max. magnitude.

* Resonant peak $M_r = \frac{1}{2\xi\sqrt{1-\xi^2}}$

max. magnitude at resonant frequency.

* Frequency domain specification valid when $\xi \leq \frac{1}{\sqrt{2}}$

$\xi > \frac{1}{\sqrt{2}}$ there no resonant peak on resonant frequency.

$$\therefore \xi = \frac{1}{\sqrt{2}} \Rightarrow \omega_r = \omega_n \sqrt{1-2\xi^2} = 0 \Rightarrow M_r = 1$$

* Bandwidth :-

$$Bw = \omega_n \sqrt{1-2\xi^2} + \sqrt{2-4\xi^2+4\xi^4}$$

range of freqⁿ at which mag. drops to -3dB i.e. $\frac{1}{\sqrt{2}}$ from the max value at the low freqⁿ.

calculation of gain margin & phase margin (T.F. given)

G.M

P.M.

- | | |
|---|---|
| ① find ω_{pc} using $\angle G(j\omega) = -180^\circ$ | ④ find ω_{qc} at $ H(j\omega) = 1$ i.e. $m=1$ |
| ② find M_{margin} (magnitude at ω_{pc}) | ③ find $\angle G(j\omega)$ (p.m.) at ω_{qc} |
| ⑤ $Gm = \frac{1}{M_{\text{margin}}}$ | |

$$Gm = -20 \log m \text{ indB}$$

* whenever plot T.F. maintain less magnitude than 1 or less at all freq. bt all the freq. range then $Gm = pm = 0$

- * In this case ω_{qc}, ω_{pc} don't exist eg. $\frac{1}{(s+5)}$
- ① $pm = \infty$ [1st priority]
 - ② None 2nd priority.
 - ③ else $\omega_{qc} = 0 \rightarrow$ calculate pm.

* ~~if $\omega_{pc} = \frac{1}{s}$~~ $\frac{1}{s} \Rightarrow \omega_{pc} = \infty \quad Gm = \infty \quad \omega_{qc} = 1 \quad pm = 90^\circ$ static

$\frac{1}{s^2} \Rightarrow \omega_{pc} = 1 \quad Gm = 1 \quad \omega_{qc} = 1 \quad pm = 0$ m.s

$\frac{1}{s^3} \Rightarrow \omega_{pc} = 0 \quad Gm = 0 \quad \omega_{qc} = 1 \quad pm = -90^\circ$ v.s

center & Radius of

$$m_{\text{circle}} = \left[\frac{m^2}{(1-m^2)}, 0 \right] \left[\left| \frac{m}{1-m^2} \right| \right]$$

$$N_{\text{circle}} = \left[-\frac{1}{2}, \frac{1}{2N} \right] \left[\frac{\sqrt{N^2+1}}{2N} \right]$$

* bandwidth (ω_b) $>$ natural freq. of oscillatⁿ (ω_n) $>$ damped osillatⁿ (ω_d) \Leftrightarrow
 $\Leftrightarrow >$ resonant freq. (ω_r)

* $G_1(s)$ has (m_1, ω_1) & $G_2(s)$ has (m_2, ω_2)

$G_1(s)$ is more stable if $m_1 > m_2$.

* If we add zero to T.F. m_m will \uparrow ω_s .

$$* m_m = \frac{k_{\text{max}}}{\text{desired value of } k}$$

most of the design work goes towards to find the system

* state-space analysis

- * state gives the future behaviour of the S/m based on present i/p and past history.
- * States variable depends on voltage across the capacitor & current through inductor. ∴ no. of state variable = sum of inductor & capacitor. ~~also~~ OR order of diff. eqn.
- * Advantages:- (see Am. state space analysis)
 - applicable to dynamic system (Linear, non-linear, TV, TIV.)
 - analysis is done by considering initial conditn.
 - More accurate than T.F.
 - gives info about controllability & observability.
- * Disadvantages → complex techniques, many computatin required.

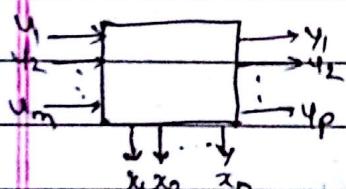
* Standard form of the state model



$$\dot{x} = Ax + Bu \rightarrow \text{state eqn}/\text{dynamic eqn}$$

$$y = Cx + Du \rightarrow \text{o/p eqn}$$

A = state matrix
 B = i/p matrix
 C = o/p matrix
 D = D_{in} matrix



$$[u]_{mx1} [x]_{nx1} [y]_{px1}$$

$$\therefore [A]_{nxn} [B]_{nxm} [C]_{pxn} [D]_{pm}$$

$$\therefore [\dot{x}]_{nx1} [y]_{px1}$$

* four forms of state models

1. Controllable canonical form
2. observable canonical form
3. diagonalization form or normal form.
4. Jordan canonical form

* We can derive or find state model from 4 types of (G(s)) given data.

* from differential eqn

2. from transfer function

3. from signal flow graph

4. from Electrical network.

* state model to diff. equatn *(by controllable canonical form)

$$1) \ddot{y} + 3\dot{y} + 5y = 9u$$

$$y = x_1, \quad \dot{y} = \dot{x}_1 = x_2, \quad \ddot{y} = \ddot{x}_1 = x_3, \quad \dot{\ddot{y}} = \dot{x}_3$$

(order = 3 \therefore 3 state space variable)

$$\therefore \dot{x}_3 = 9u - 7x_1 - 5x_2 - 3x_3$$

$$\begin{array}{l} \text{① } \dot{x}_1 \\ \text{② } \dot{x}_2 \\ \text{③ } \dot{x}_3 \end{array} = \begin{bmatrix} 0 & 1 & 0 \\ 0 & 0 & 1 \\ -7 & -5 & -3 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \\ 9 \end{bmatrix} u \quad \left. \begin{array}{l} \text{state eqn} \\ \text{B}_{CCF} \\ \text{C}_{CCF} \end{array} \right\}$$

$$\begin{bmatrix} y \\ \dot{y} \end{bmatrix} = \begin{bmatrix} 1 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 1 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} \quad \left. \begin{array}{l} \text{A}_{CCF} \\ \text{D}_{CCF} \end{array} \right\} \text{Olp. eqn}$$

* (by observable canonical form) conversions.

$$\begin{array}{l} \text{Accf} \\ \text{B}_{CCF} \\ \text{C}_{CCF} \\ \text{are in} \\ \text{above} \\ \text{eg.} \end{array} \left\{ \begin{array}{l} \underline{A_{CCF}} = (\underline{A_{CCF}})^T \\ \underline{B_{CCF}} = \begin{bmatrix} b_1 \\ b_2 \\ b_3 \\ b_4 \end{bmatrix} \uparrow \text{where } B_{CCF} = \begin{bmatrix} b_1 \\ b_2 \\ b_3 \\ b_4 \end{bmatrix} \\ \underline{C_{CCF}} = [c_1 \ c_2 \ c_3 \ c_4] \quad \text{when } C_{CCF} = [c_1 \ c_2 \ c_3 \ c_4] \end{array} \right.$$

* state model eqn to the transfer f^n.

$$y(s) = \frac{x_1}{s^3 + 5s^2 + 6s + 25 + 3s} \quad \text{or} \quad s^3 = x_3$$

$$u(s) = s^2 + 5s + 6 \quad \therefore u = \dot{x}_2 + 5x_2 + x_1 \Rightarrow \dot{x}_2 = u - 5x_2 - 6x_1$$

$$\dot{x}_2 = 2x_2 + 3x_1, \quad \dot{x}_1 = x_2$$

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \end{bmatrix} = \begin{bmatrix} 0 & 1 \\ -6 & -5 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \end{bmatrix} + \begin{bmatrix} 0 \\ 1 \end{bmatrix} u$$

$$y = [3 \ 2] \begin{bmatrix} x_1 \\ x_2 \end{bmatrix} \quad \text{(with some sign)}$$

common factor in the numerator

* diagonalization form.

$$y(s) = \frac{1}{u(s)(s+1)(s+2)(s+3)} \Rightarrow \frac{1}{u(s)} = \frac{1}{2} \frac{1}{(s+1)} - \frac{1}{(s+2)} + \frac{1}{2} \frac{1}{(s+3)}$$

$$(Y)^T = (1 \ 1 \ 1) \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} \quad \text{or} \quad y(s) = x_1 + x_2 + x_3$$

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \dot{x}_3 \end{bmatrix} = \begin{bmatrix} -1 & 0 & 0 \\ 0 & -2 & 0 \\ 0 & 0 & -3 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} + \begin{bmatrix} \frac{1}{2} \\ -1 \\ \frac{1}{2} \end{bmatrix} u$$

Poles/eigen values. Partial fraction

$$\begin{array}{l} \text{① } x_1 = \frac{1}{2} u \\ \text{② } x_2 = -4 - 2x_2 \\ \text{③ } x_3 = \frac{1}{2} u - 3x_3 \end{array}$$

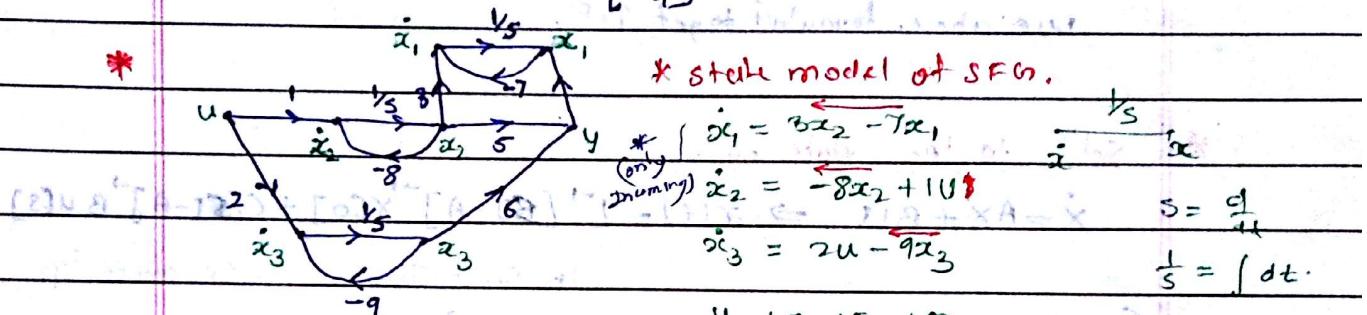
$\frac{dx}{dt}$

* Jordan canonical form

$$Y(s) = \frac{1}{(s+5)^3(s+10)} = \frac{A'}{(s+5)} + \frac{B'}{(s+5)^2} + \frac{C'}{(s+5)^3} + \frac{D'}{(s+10)}$$

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \dot{x}_3 \\ \dot{x}_4 \end{bmatrix} = \begin{bmatrix} -5 & 1 & 0 & 0 \\ 0 & -5 & 1 & 0 \\ 0 & 0 & -5 & 0 \\ 0 & 0 & 0 & -10 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \\ x_4 \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \\ 0 \\ 1 \end{bmatrix} u$$

$$\boxed{y} = \begin{bmatrix} A' & B' & C' & D' \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \\ x_4 \end{bmatrix} + \boxed{A - 5I}$$



$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \dot{x}_3 \end{bmatrix} = \begin{bmatrix} -7 & 3 & 0 \\ 0 & -8 & 0 \\ 0 & 0 & 9 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} + \begin{bmatrix} 0 \\ 1 \\ 2 \end{bmatrix} u \quad y = \begin{bmatrix} 2 & 5 & 6 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix}$$

* State model of Electrical N/W.

* Select state variable as

voltage across capacitor and current through inductor.

* Total state variable = sum of Inductor & Capacitor

* write independent KVL & KCL eqns. S.V = V_2, I_4, I_L

① KCL $I_C = I_{L1} + I_{L2} \Rightarrow C \frac{dv_c}{dt} = I_{L1} + I_{L2} \Rightarrow i_c = I_{L1} + \frac{I_{L2}}{C} \quad \text{--- (1)}$

② KVL $V_i = R_1 I_{L1} + L_1 \frac{dI_{L1}}{dt} + V_C \quad \text{--- (2)}$

③ KVL $V_o = L_2 \frac{dI_{L2}}{dt} + I_{L2} R_2 \Rightarrow i_{L2} = \frac{V_o}{L_2} - \frac{I_{L2} R_2}{L_2} \quad \text{--- (3)}$

$$V_o = I_{L2} R_2 \quad \text{--- (4)}$$

$$\therefore \begin{bmatrix} v_c \\ i_{L1} \\ i_{L2} \end{bmatrix} = \begin{bmatrix} 0 & 1/C & -1/C \\ -R_1 & -L_1 & 0 \\ 1/L_2 & 0 & -R_2 \end{bmatrix} \begin{bmatrix} v_c \\ i_{L1} \\ i_{L2} \end{bmatrix} + \begin{bmatrix} 0 \\ 1/L_1 \\ 0 \end{bmatrix} v_i$$

state eqn
from (1)(2)(3)

$$V_o [0 \ 0 \ R_2] \begin{bmatrix} v_c \\ i_{L1} \\ i_{L2} \end{bmatrix} = R_2 v_i \quad \text{from (4)}$$

* Trans for f^n from state model.

$$T.F = C \frac{\text{Adj}[S\mathbf{I} - A]}{B} + D. \quad (1)$$

$| S\mathbf{I} = A |$

$$* \begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \end{bmatrix} = \begin{bmatrix} -2 & -3 \\ 4 & 2 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \end{bmatrix} + \begin{bmatrix} 3 \\ 5 \end{bmatrix}$$

$$[y] = [1 \ 1] \begin{bmatrix} x_1 \\ x_2 \end{bmatrix}$$

$$[S\mathbf{I} - A] = \begin{bmatrix} s+2 & 3 \\ -4 & s-2 \end{bmatrix} \xrightarrow{D=0} \text{Adj } A = \begin{bmatrix} s-2 & -3 \\ 4 & (s+2) \end{bmatrix}$$

use above formula (1) to get T.F.

* Solⁿ to the state eqn.

$$\dot{x} = Ax + Bu \Rightarrow x(t) = \underbrace{L^{-1}[S\mathbf{I} - A]^{-1}x(0)}_{ZIR \text{ due to IC}} + \underbrace{C[S\mathbf{I} - A]^{-1}Bu(s)}_{ZSR \text{ due to i/p}}$$

(Zero i/p Response due to initial condtn)

$$x(t) = \underbrace{x(0) e^{At}}_{ZIR} + \underbrace{\int_0^t e^{A(t-T)} Bu(T) dT}_{ZSR}$$

comparing ZIR.

$$\phi(t) = e^{At} = L^{-1}[S\mathbf{I} - A]^{-1} \rightarrow \underline{\text{STM state transition Mx}}$$

$$[S\mathbf{I} - A]^{-1} = L[\phi(t)]$$

comparing ZSR.

$$\int_0^t \phi(t-T) Bu(T) dT = L^{-1}[\phi(s) Bu(s)]$$

$$\therefore x(t) = e^{At} x(0) + L^{-1}[\phi(s) * Bu(s)]$$

Properties of STM.

$$① \phi(0) = I \text{ (Identity mx)} \quad ② \phi^k(t) = [e^{At}]^k = e^{Akt}$$

$$③ \phi^{-1}(t) = \phi(-t) \quad ④ \phi(t_1, t_2) = \phi(t_1) \cdot \phi(t_2)$$

$$⑤ \phi(t_2 - t_1) \cdot \phi(t_1 - t_0) = \phi(t_2 - t_0)$$

If state model is homogeneous solⁿ is ZIR

If state model is Non-homogeneous solⁿ is ZIR + ZSR.

* Controllability & observability.

- If it is possible to transfer the initial states to the desired state in a finite time period by the control i/p, then it is controllable.

e.g. in SF6, from ip to cash and every state variable if path is exist then s/m is controllable else not.

- Kalman's test for controllability $\mathbf{Q}_c = [C^T \ A\ C \ A^2\ C \ \dots \ A^{n-1}C]$
Rank of \mathbf{Q}_c = Rank of A $\Rightarrow |Q_c| \neq 0$

- A s/m is said to be observable if it is possible to determine initial states of the system by observing the o/p in a finite time interval.

(Kalman's test for observability) $\mathbf{Q}_o = [C^T \ A^T C^T \ (A^T)^2 C^T \ \dots \ (A^T)^{n-1} C^T]$

$$\text{Rank } \mathbf{Q}_o = \text{Rank } A = \begin{bmatrix} C \\ CA \\ CA^2 \\ \vdots \\ CA^{n-1} \end{bmatrix}$$

$|Q_o| \neq 0$.

- The pole/zero cancellation makes system non

- uncontrollable but observable
- controllable but unobservable
- Uncontrollable & unobservable
but not makes controllable & observable.

- Gibson's test for controllability & observability.

$$\dot{x}(t) = \begin{bmatrix} -2 & 0 \\ 0 & -3 \end{bmatrix} x(t) + \begin{bmatrix} 0 & 2 \\ 1 & -3 \end{bmatrix} u(t)$$

controllable
non zero.

$$y(t) = \begin{bmatrix} 2 & 3 \\ 0 & -1 \end{bmatrix} x(t)$$

non zero, observable.

* Compensators & controllers

* Compensators:

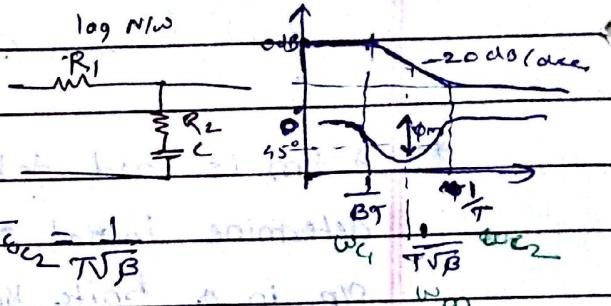
* Lag compensator:

→ A compensator having the characteristic of lag network is called a lag compensator.

→ It result in large improvement in steady state performance but results in slower response due to reduced B.W.

→ It is essentially 1PF.

$$T.F. = \frac{s+2\zeta}{s+\frac{P}{C}} = \frac{s+2\zeta}{s+\frac{1}{CT}}$$



$$\therefore \text{max phase lag} = \phi_m = \tan^{-1} \frac{1-\beta}{2\sqrt{\beta}}$$

$$\therefore \text{max log angle} = \phi_m = \tan^{-1} \frac{1-\beta}{2\sqrt{\beta}}$$

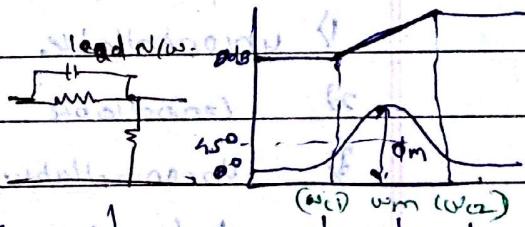
* Lead compensator

→ A compensator having characteristics of lead N/wk.

→ It appreciably improves the transient response. It increases the B.W. & improves speed of the response, also reduces amount of overshoot.

→ basically 4PF.

$$T.F. = \frac{s+2\zeta}{s+\frac{P}{C}} = \frac{s+2\zeta}{s+\frac{1}{CT}}$$



$$\therefore \text{max phase lag} = \phi_m = \tan^{-1} \frac{1-\alpha}{2\sqrt{\alpha}}$$

$$\therefore \text{max phase lead angle} \Rightarrow \phi_m = \tan^{-1} \left(\frac{1-\alpha}{2\sqrt{\alpha}} \right); \alpha = \frac{1-s\omega_m}{1+s\omega_m \phi_m}$$

* Lag lead compensator:

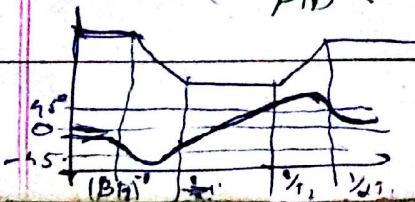
→ having characteristics of both lag & lead N/w.

→ It improves both transient & steady state response.

$$T.F. = (s+\frac{1}{T_1})(s+\frac{1}{T_2})$$

$$(s+\frac{1}{\beta T_1})(s+\frac{1}{\alpha T_2}) \quad [\beta > 1; 0 < \alpha < 1]$$

$$\begin{matrix} * & * & * & * \\ -1 & -1 & 1 & 1 \\ \alpha T_2 & \beta T_2 & T_1 & \beta T_1 \end{matrix}$$



* controllers:

① Proportional controller $G_c(s) = K_p$.

→ used to vary the transient response, it is usually damp with gain K_p .

② Integral controller $G_c(s) = K_i/s$

→ used to inc. s.s. error by type of system.

→ but dis-adv. → stability ↓/scs.

③ Derivative controller $G_c(s) = K_d s$

→ used to inc. stability by adding zeros

→ but dis-adv. → s.s. error ↑/scs. ∵ type of system ↓/scs.

④ (P-I) controller (Proportion-Integral) $\Rightarrow G_c(s) = K_p + \frac{K_i}{s}$

→ used to inc. ss.error without effecting stability ∵ poles are at origin & zero is added.

→ Type of s/m 1/scs ∵ ss.error ↑/scs.

→ IT acts as LPF. B.W. ↓/scs and τ_p ↑/scs (∴ transient response becomes slow)

→ & improves basic reducing max overshoot.

→ filters out high frequency noise.

⑤ P-D controller $G_c(s) = K_p s + K_d$

→ used to improve stability without effecting s.s. error.

∴ type is not changed & zero is added.

→ improving ζ_p & Reducing max overshoot.

→ PD is HPF. Hence B.W. ↑/scs, τ_p , τ_i , τ_d ↓/scs (transient time response improved)

→ improves ζ_p , M_p & M_r

→ Noise enters at high frequency.

→ Possibly requiring relatively large capacitor in circuit.

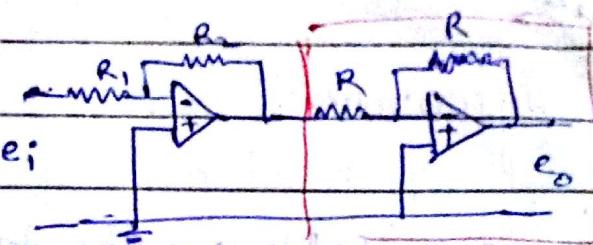
⑥ PID controller $G_c(s) = K_p + \frac{K_i}{s} + K_d s$

It is used to inc. ζ_p & inc. stability ∵ pole at origin zero added.

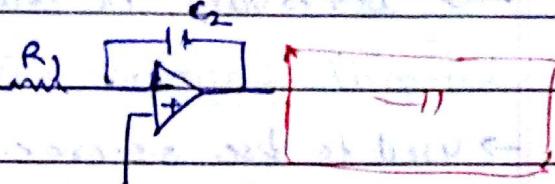
* Opamp circuit

* Control action

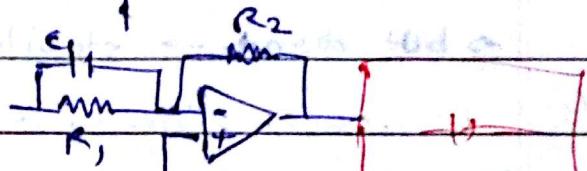
$$P \quad * G(s) = \frac{E_o(s)}{E_i(s)} = \frac{R_2}{R_1}$$



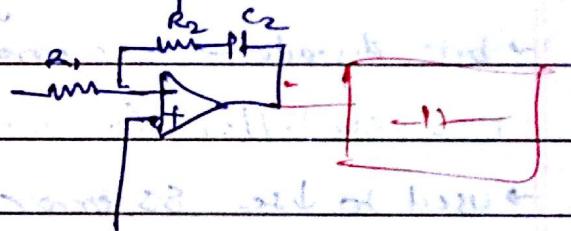
I $\frac{1}{R_1 C_2 s}$



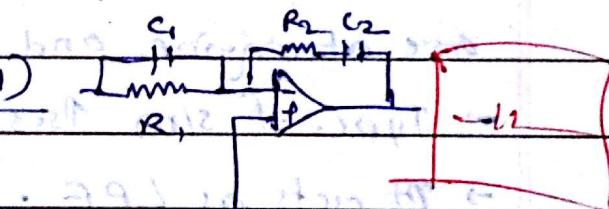
PD $\frac{R_2}{R_1} (R_1 C_1 s + 1)$



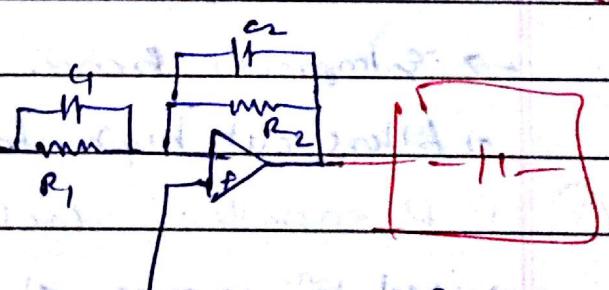
PI $\frac{R_2}{R_1} \frac{R_2 C_2 s + 1}{R_2 C_2 s}$



PIP $\frac{R_2}{R_1} \frac{(R_1 C_1 s + 1)(R_2 C_2 s + 1)}{R_2 C_2 s}$



Lead
lag $\frac{R_2 (R_1 C_1 s + 1)}{R_1 (R_2 C_2 s + 1)}$



Lag-lead $\frac{R_3 [(R_1 + R_2) C_1 s + 1]}{R_3 [(R_1 C_1 s + 1) [(R_2 + R_3) C_2 s + 1]]}$

