Defining the physical sources of sour Non-radiating filter desig Sources of sound in an axi-symmetric j Conclusio

# Separating propagating and non-propagating dynamics in fluid-flow equations

Samuel Sinayoko,

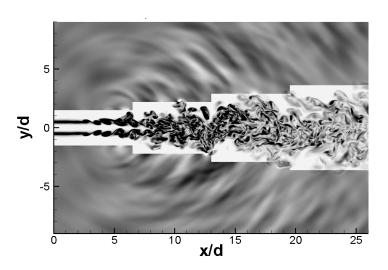
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Institute of Sound and Vibration Research

May 2009



Institute of Sound and
Vibration Research
Sources of sound in an axi-symmetric jet
Conclusion



#### Introduction

How to define the physical sources of sound?

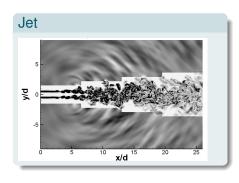
#### **Objectives**

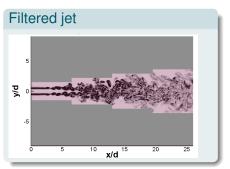
- Derive an expression for the physical sources of sound.
- ② Demonstrate that it is possible to separate the radiating and non-radiating parts of the flow.
- Ompute the physical sources of sound.

### Outline

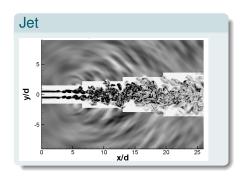
- Defining the physical sources of sound
  - Goldstein's theory
  - Equations
- 2 Non-radiating filter design
  - Problem description
  - Filter defining properties
  - Local filter
  - Global filter
- Sources of sound in an axi-symmetric jet
  - Flow description
  - Filter design
  - Sound sources

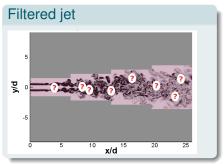
### Goldstein's theory



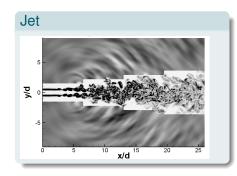


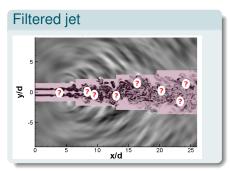
### Goldstein's theory





### Goldstein's theory





These sources should be close to the true sources of sound.

## Governing equation for fluctuating quantities

#### Flow filtering

$$\mathcal{L}f = \overline{f} \tag{1}$$

#### Flow decomposition

$$f=\overline{f}+f' \tag{2}$$

## Governing equation for fluctuating quantities

#### Flow filtering

$$\mathcal{L}f = \overline{f} \tag{1}$$

#### Flow decomposition

$$f = \overline{f} + f' \tag{2}$$

#### Conservation of mass

$$\frac{\partial \rho}{\partial t} + \frac{\partial \rho \mathbf{v}_j}{\partial \mathbf{x}_i} = \mathbf{0},\tag{3}$$

$$\frac{\partial \overline{\rho}}{\partial t} + \frac{\partial \overline{\rho v_j}}{\partial x_i} = 0. \tag{4}$$

#### Flow filtering

$$\mathcal{L}f = \overline{f} \tag{1}$$

#### Flow decomposition

$$f = \overline{f} + f' \tag{2}$$

Conservation of mass

$$\frac{\partial \rho}{\partial t} + \frac{\partial \rho v_j}{\partial x_j} = 0, \tag{3}$$

$$\frac{\partial \overline{\rho}}{\partial t} + \frac{\partial \overline{\rho} v_j}{\partial x_i} = 0.$$
 (4)

Conservation of mass for fluctuating quantities

$$\frac{\partial \rho'}{\partial t} + \frac{\partial (\rho v_j)'}{\partial x_i} = 0.$$

## Governing equation for fluctuating quantities

Conservation of mass for fluctuating quantities

$$\frac{\partial \rho'}{\partial t} + \frac{\partial (\rho v_j)'}{\partial x_j} = 0.$$
 (5)

Momentum conservation for fluctuating quantities

$$\frac{\partial(\rho v_i)'}{\partial t} + \frac{\partial(\rho v_i v_j)'}{\partial x_i} + \frac{\partial p'}{\partial x_i} = \frac{\partial \sigma'_{ij}}{\partial x_i}.$$
 (6)

Conservation of mass for fluctuating quantities

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 (6)

Taking  $\partial(6)/\partial x_i - \partial(5)/\partial t$  gives

$$\frac{\partial^2 p'}{\partial x_i x_i} - \frac{\partial^2 \rho'}{\partial t^2} + \frac{\partial^2 (\rho v_i v_j)'}{\partial x_i \partial x_j} = \frac{\partial^2 \sigma'_{ij}}{\partial x_i \partial x_j}.$$
 (7)

## Governing equation for fluctuating quantities

Favre averaging, 
$$\tilde{f} = \overline{\rho f}/\overline{\rho}$$
, (8)

#### Governing equation

$$\frac{\partial^{2} \rho'}{\partial x_{i} \partial x_{i}} - \frac{\partial^{2} \rho'}{\partial t^{2}} + \frac{\partial^{2}}{\partial x_{i} \partial x_{j}} (\tilde{v}_{i} \tilde{v}_{j} \rho' + \overline{\rho} \tilde{v}_{j} v'_{i} + \overline{\rho} \tilde{v}_{i} v'_{j}) = \frac{\partial^{2} \sigma_{ij'}}{\partial x_{i} \partial x_{j}} + s$$
 (9)

## Governing equation for fluctuating quantities

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#### Governing equation

$$\frac{\partial^{2} \rho'}{\partial x_{i} \partial x_{i}} - \frac{\partial^{2} \rho'}{\partial t^{2}} + \frac{\partial^{2}}{\partial x_{i} \partial x_{j}} (\tilde{v}_{i} \tilde{v}_{j} \rho' + \overline{\rho} \tilde{v}_{j} v'_{i} + \overline{\rho} \tilde{v}_{i} v'_{j}) = \frac{\partial^{2} \sigma_{ij'}}{\partial x_{i} \partial x_{j}} + s$$
 (9)

#### Source definition

$$s = -\frac{\partial^2}{\partial x_i \partial x_j} \left( T_{ij} + \rho v_i' v_j' + \tilde{v}_i \rho' v_j' + \tilde{v}_j \rho' v_i' \right)$$

$$T_{ij} = -\overline{\rho} (\widetilde{v_i v_j} - \tilde{v}_i \tilde{v}_j).$$
(10)

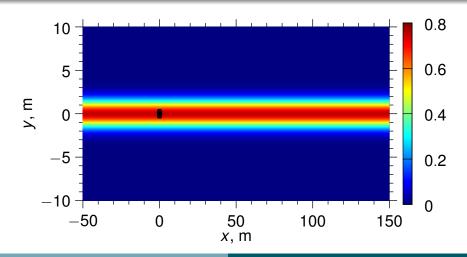
$$T_{ii} = -\overline{\rho}(\widetilde{v_i}\widetilde{v_i} - \widetilde{v_i}\widetilde{v_i}). \tag{11}$$

### Outline

- Defining the physical sources of sound
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  - Filter defining properties
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  - Sound sources

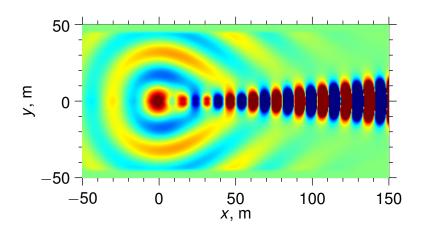
## Problem description

Parallel flow



## Problem description

Pressure field



Filter defining properties

### Defining properties

#### Fourier transform

$$f(\mathbf{x},t) \to F(\mathbf{k},\omega)$$

$$f(\mathbf{x},t) \to F(\mathbf{k},\omega)$$
  
 $\overline{f}(\mathbf{x},t) \to \overline{F}(\mathbf{k},\omega)$ 

Filter defining properties

## Defining properties

#### Fourier transform

$$f(\mathbf{x},t) \to F(\mathbf{k},\omega)$$

$$\overline{f}(\mathbf{x},t) \to \overline{F}(\mathbf{k},\omega)$$

#### Non-radiating condition

$$\overline{F}(\mathbf{k},\omega) = 0$$
 for  $|\mathbf{k}| = \frac{|\omega|}{c_{\infty}}$ 

## **Defining properties**

#### Fourier transform

$$f(\mathbf{x},t) \to F(\mathbf{k},\omega)$$

$$\overline{f}(\mathbf{x},t) \to \overline{F}(\mathbf{k},\omega)$$

#### Non-radiating condition

$$\overline{F}(\mathbf{k},\omega) = 0$$
 for  $|\mathbf{k}| = \frac{|\omega|}{G_{\infty}}$ 

#### Additional requirement

$$\overline{F}(\mathbf{k},\omega) = F(\mathbf{k},\omega) \quad \text{for} \quad |\mathbf{k}| \neq \frac{|\omega|}{G_{\infty}}$$

Vibration Research Sources of sound in an axi-symmetric jet

Local filter

## Local filter

#### Filter definition

#### D'Alembertian filter

$$\overline{f}(\mathbf{x},t) = \left(\frac{1}{c_{\infty}^2} \frac{\partial^2}{\partial t^2} - \nabla^2\right) f(\mathbf{x},t),$$

Local filter

#### Local filter Filter definition

#### D'Alembertian filter

$$\overline{f}(\mathbf{x},t) = \left(\frac{1}{c_{\infty}^2} \frac{\partial^2}{\partial t^2} - \nabla^2\right) f(\mathbf{x},t),$$

#### Frequency domain

$$\overline{F}(\mathbf{k},\omega) = \left( |\mathbf{k}|^2 - \frac{\omega^2}{c_{\infty}^2} \right) F(\mathbf{k},\omega)$$

Local filter

#### Local filter Filter definition

#### D'Alembertian filter

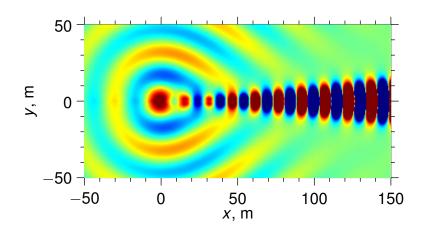
$$\overline{f}(\mathbf{x},t) = \left(\frac{1}{c_{\infty}^2} \frac{\partial^2}{\partial t^2} - \nabla^2\right) f(\mathbf{x},t),$$

#### Frequency domain

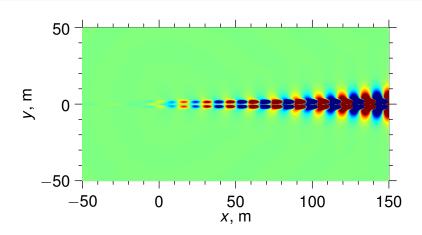
$$\overline{F}(\mathbf{k},\omega) = \left( |\mathbf{k}|^2 - \frac{\omega^2}{c_{-2}^2} \right) F(\mathbf{k},\omega)$$

$$\Rightarrow \overline{F}(\mathbf{k},\omega) = 0$$
 for  $|\mathbf{k}| = \frac{|\omega|}{c_{\infty}}$ 

## Local filter Results



## Local filter



### Global filter

Filter definition

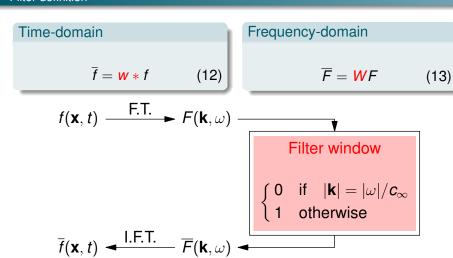
$$\bar{f} = \mathbf{w} * f \tag{12}$$

$$\overline{F} = WF$$
 (13)

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## Global filter

Filter definition



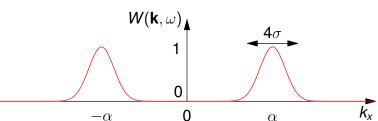
## Global filter

#### Filter definition

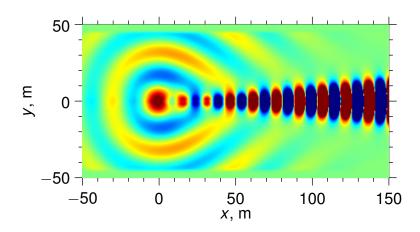
#### Gaussian filter

$$W(\mathbf{k},\omega) = \exp\left(-rac{(k_{X}-lpha)^{2}}{2\sigma^{2}}
ight) + \exp\left(-rac{(k_{X}+lpha)^{2}}{2\sigma^{2}}
ight)$$

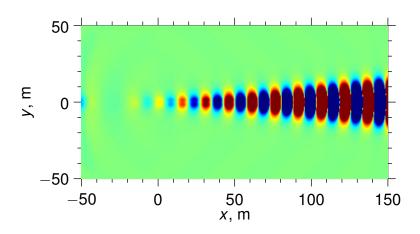
$$\alpha = 0.68 \text{m}^{-1}, \quad \sigma = 0.1 \text{m}^{-1}.$$



## Global filter Results

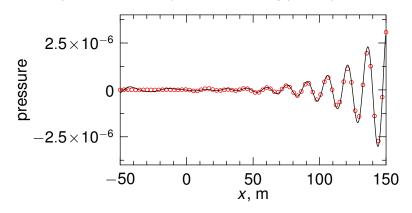


## Global filter Results



## Global filter

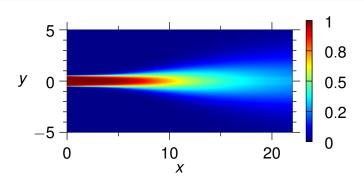
#### Comparison with analytical result along profile y = 15m



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## Flow description



Mean flow excited at two frequencies:

$$\omega_1 = 2.2$$
,

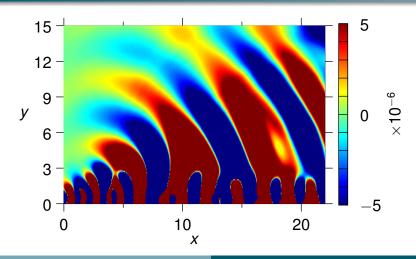
$$\omega_2 = 3.4$$
,

$$\Delta\omega = 1.2$$
.

Flow description Filter design Sound sources

## Flow description

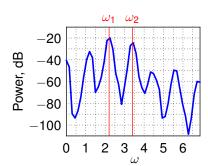
Pressure field



Flow description Filter design Sound sources

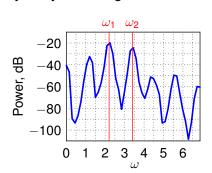
## Flow description Frequency analysis

#### Hydrodynamic region

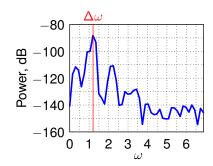


## Flow description Frequency analysis

#### Hydrodynamic region

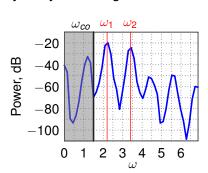


#### Acoustic region

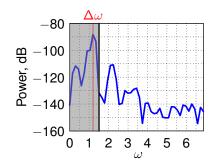


# Flow description Frequency analysis

### Hydrodynamic region



### Acoustic region

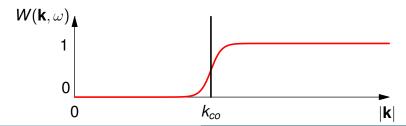


# Filter design Definition

### Tanh filter

$$W(\mathbf{k},\omega) = rac{1}{2} \left[ 1 + anh \left( rac{|\mathbf{k}| - k_{co}}{\sigma} 
ight) 
ight],$$

$$k_{co} = 1.3, \quad \sigma = 0.2.$$



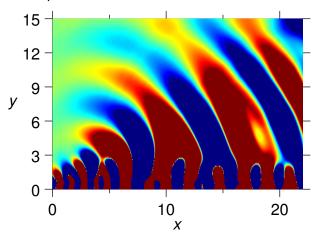
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# Filter design

### Validation

### Pressure field p

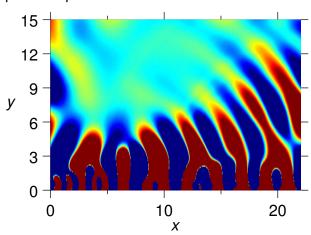


Flow description
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Sound sources

# Filter design

Validation

### Filtered pressure $\overline{p}$



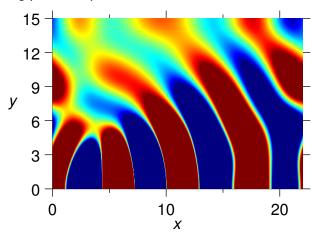
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# Filter design

### Validation

### Fluctuating pressure p'

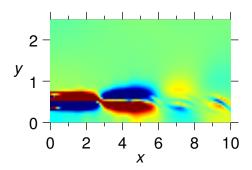


Flow description
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# Sound sources

Using non-radiating filter

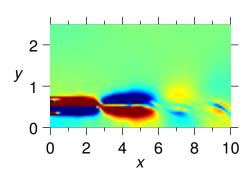
### Sound source s



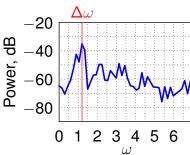
Sound sources

### Sound sources Using non-radiating filter

### Sound source s



## Spectrum at (4.0, 0.55)

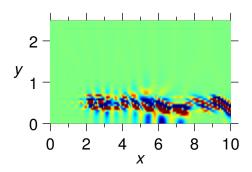


Flow descriptio Filter design Sound sources

### Sound sources

Using time average filter

### Sound source s

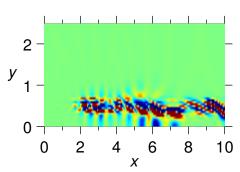


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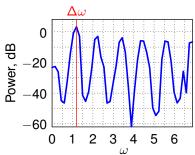
## Sound sources

Using time average filter

### Sound source s



### Spectrum at (5.5, 0.5)



### Sound sources

Evolution in time

(source)

### Conclusion and future work

### Results

- Sound source definition
- Separation possible with convolution filters.
- Clearer physical interpretation of the sources.

### Conclusion and future work

### Results

- Sound source definition
- Separation possible with convolution filters.
- Clearer physical interpretation of the sources.

### Future work

- Mixing-layer and a two-dimensional jet.
- Physical mechanism behind the sound sources.



Defining the physical sources of sounce

Non-radiating filter design
Sources of sound in an axi-symmetric jet

Conclusion

# Acknowledgements



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