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HW4

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Prob 1:

The first two eqs in ω_1 & ω_2 can be written as follow

$$\omega_1 = (\dot{\phi} \sin \theta) \sin \psi + \dot{\theta} (\cos \psi)$$

$$\omega_2 = (\dot{\phi} \sin \theta) \cos \psi + \dot{\theta} (-\sin \psi)$$

$$\begin{bmatrix} \omega_1 \\ \omega_2 \end{bmatrix} = \begin{bmatrix} \cos \psi & \sin \psi \\ -\sin \psi & \cos \psi \end{bmatrix} \begin{bmatrix} \dot{\theta} \\ \dot{\phi} \sin \theta \end{bmatrix}$$

orthonormal Rot Matrix
the inverse is transpose

$$\therefore \begin{bmatrix} \dot{\theta} \\ \dot{\phi} \sin \theta \end{bmatrix} = \begin{bmatrix} \cos \psi & -\sin \psi \\ \sin \psi & \cos \psi \end{bmatrix} \begin{bmatrix} \omega_1 \\ \omega_2 \end{bmatrix}$$

$$\boxed{\dot{\theta} = \cos \psi \omega_1 - \sin \psi \omega_2}$$

$$\dot{\phi} \sin \theta = \sin \psi \omega_1 + \cos \psi \omega_2$$

$$\boxed{\dot{\phi} = \frac{\sin \psi \omega_1 + \cos \psi \omega_2}{\sin \theta}}$$

Singularity when
 $\theta = 0 \rightarrow \sin \theta \rightarrow 0$

From the third Eq

$$\dot{\psi} = \omega_3 - \dot{\phi} \cos \theta$$

$$\dot{\psi} = \omega_3 - \frac{\sin \psi \omega_1 + \cos \psi \omega_2}{\tan \theta}$$

Also affected
by singularity
at $\theta = 0$

Prob 2: The KE of the Element dm is

$$dT = \frac{1}{2} \vec{v} \cdot \vec{v} dm$$

where $\int dm = M$

$$\int_{\text{body}} dT = T = \frac{1}{2} \int_{\text{body}} \vec{v} \cdot \vec{v} dm$$

$$\vec{v} = \vec{v}_{\text{com}} + \vec{\omega}^{bi} \times \vec{r}$$

$$\vec{v} \cdot \vec{v} = (\vec{v}_{\text{com}} + \vec{\omega}^{bi} \times \vec{r}) \cdot (\vec{v}_{\text{com}} + \vec{\omega}^{bi} \times \vec{r})$$

$$= \vec{v}_{\text{com}} \cdot \vec{v}_{\text{com}} + 2 \vec{v}_{\text{com}} \cdot (\vec{\omega}^{bi} \times \vec{r}) + (\vec{\omega}^{bi} \times \vec{r}) \cdot (\vec{\omega}^{bi} \times \vec{r})$$

$$\therefore T = \frac{1}{2} \int \vec{v}_{\text{com}} \cdot \vec{v}_{\text{com}} dm + \int \vec{v}_{\text{com}} \cdot (\vec{\omega}^{bi} \times \vec{r}) dm + \frac{1}{2} \int (\vec{\omega}^{bi} \times \vec{r}) \cdot (\vec{\omega}^{bi} \times \vec{r}) dm$$

\vec{v}_{com} & $\vec{\omega}^{bi}$ are not functions of positions on Rigid body

$$T = \frac{1}{2} \vec{v}_{\text{com}} \cdot \vec{v}_{\text{com}} \int dm + \vec{v}_{\text{com}} \cdot \left(\vec{\omega}^{bi} \times \int \vec{r} dm \right) + "$$

$$T = \frac{1}{2} M (\vec{v}_{\text{com}} \cdot \vec{v}_{\text{com}}) + \vec{v}_{\text{com}} \cdot \left(\vec{\omega}^{bi} \times \vec{0} \right) + \frac{1}{2} \vec{\omega}^{bi} \cdot \mathcal{I} \vec{\omega}^{bi}$$

The Integral
Evaluates to the definition
of COM, as \vec{r} is dist
from COM, it is simply $\vec{0}$

from
lecture
notes
def of \mathcal{I}

$$T = \frac{1}{2} M (\vec{v}_{\text{com}} \cdot \vec{v}_{\text{com}}) + \frac{1}{2} \vec{\omega}^{bi} \cdot \mathcal{I} \vec{\omega}^{bi}$$

Prob 3. a

$$I_{\text{Principal}} = \begin{bmatrix} 1.29 & 0 & 0 \\ 0 & 9.68 & 0 \\ 0 & 0 & 10.10 \end{bmatrix} \times 10^6 \text{ Kg-m}^2 = \begin{bmatrix} A & 0 & 0 \\ 0 & B & 0 \\ 0 & 0 & C \end{bmatrix}$$

→ about b_1
→ about b_2
→ about b_3

The stable configuration finally settles to $C > B > A$ (Already true in our case), so the axes, $b_1, b_2, \& b_3$ will be aligned along.

$b_1 \rightarrow (\text{YAW}) \rightarrow$	aligned with radius vector.
$b_2 \rightarrow (\text{ROLL}) \rightarrow$	aligned with velocity vector
$b_3 \rightarrow (\text{PITCH}) \rightarrow$	in the direction of orbit normal

Prob 3. b

The Eqs of Motion for a body in orbit under Torque-Free Gravity Gradient are

$$A\ddot{\psi}_1 + (C-B-A)\Omega\dot{\psi}_2 + (C-B)\Omega^2\psi_1 = 0 \quad (\text{yaw})$$

$$B\ddot{\psi}_2 + (B+A-C)\Omega\dot{\psi}_1 + 4(C-A)\Omega^2\psi_2 = 0 \quad (\text{roll})$$

$$C\ddot{\psi}_3 + 3(B-A)\Omega^2\psi_3 = 0 \quad (\text{pitch})$$

where ψ_1, ψ_2, ψ_3 are rotations about axes b_1, b_2, b_3 respectively

Where, $\Omega^2 = \frac{\mu}{R^3}$, can be computed from the time period of the orbit as follows

$$\Omega = 2\pi f = \frac{2\pi}{T} \rightarrow T = 90 \text{ min}$$

To find frequencies we need to find the System Poles (or Eigenvalues, or roots of characteristic Eq)

1) For Pitch: it is a Spring-Mass System form. like $m\ddot{x} + kx = 0$ where $\omega_n = \sqrt{\frac{k}{m}}$

$$\omega_{n, \text{pitch}} = \Omega \sqrt{\frac{3(B-A)}{C}} = 0.001836 \text{ Rad/sec}$$

2) For Roll & Yaw it's a Coupled System of two 2nd order linear ODE's, can be written as state-space representation of $\dot{X} = AX$, where

$X^T = [\psi_1 \ \psi_2 \ \dot{\psi}_1 \ \dot{\psi}_2]$, then A becomes

$$A = \begin{bmatrix} 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 1 \\ -(C-B)\Omega^2 & 0 & 0 & -(C-B-A)\Omega \\ 0 & -4\frac{(C-A)}{B}\Omega^2 & -\frac{(B+A-C)}{B}\Omega & 0 \end{bmatrix}$$

The poles of the Roll & Yaw System are
Eigenvalues of A (the Imaginary part
of it is the freq)

$$\text{Eg. } \lambda_{1,2} = -\underbrace{\frac{1}{\tau}}_{\substack{\text{Real part} \\ \text{determines} \\ \text{the stability of the system}}} \pm i \underbrace{\omega_d}_{\substack{\text{damped Natural} \\ \text{freq}}}$$

The Eigenvalues of A

$$\lambda_{1,2} = \pm i 6.5794 \times 10^{-4} \text{ rad/sec}$$

$$\lambda_{3,4} = \pm i 0.002240 \text{ rad/sec}$$

So the Final three frequencies are

$$6.5794 \times 10^{-4} \text{ Rad/sec}$$

$$0.00224023 \text{ Rad/sec}$$

$$0.00183682 \text{ Rad/sec} \rightarrow \text{Pitch}$$

} Roll & Yaw

The numerical results are Evaluated
using MATLAB Script that is attached
within the pdf Document.