

EN 203 > problem set #3 solutions

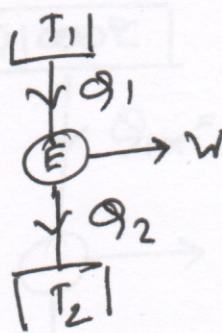
1) $\{ Q_1 = 85 \text{ kJ} \}$ given data
 $W = 21 \text{ kJ} \}$

$$\eta = \frac{W}{Q_1} = \frac{21}{85} = 0.247$$

and also, $\eta = 0.247 = 1 - \frac{Q_2}{Q_1}$

$$\therefore \frac{Q_2}{Q_1} = 1 - 0.247 = 0.753$$

$$\therefore Q_2 = 0.753 Q_1 = 64 \text{ kJ} \quad \text{(Ans)}$$



OR

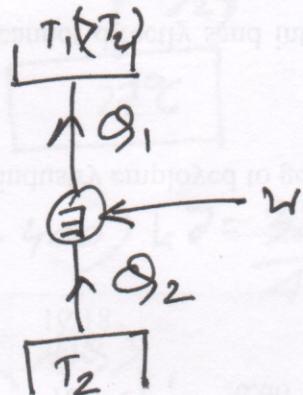
$$Q_1 - Q_2 = W \quad \therefore Q_2 = Q_1 - W = 64 \text{ kJ} \quad \text{(Ans)}$$

2) $W = 75 \text{ kJ} \}$ given
 $Q_2 = 220 \text{ kJ} \}$

From 1st law of TD: $Q_1 - Q_2 = W$

$$\therefore Q_1 = W + Q_2$$

$$\therefore Q_1 = (75 + 220) \text{ kJ} = 295 \text{ kJ}$$



~~COP_{Ref}~~ ~~cooling effect~~ ~~Work done~~

$$\text{COP}_{\text{Ref}} = \frac{\text{cooling effect}}{\text{work input}} = \frac{220}{75} = 2.93 \quad \text{Ans}$$

$$\text{COP}_{\text{H.P.}} = \frac{\text{heating effect}}{\text{work input}} = \frac{295}{75} = 3.93$$

OR

$$\text{COP}_{\text{H.P.}} = 1 + \text{COP}_{\text{Ref}} = 3.93$$

3) Applying the 1st law of TD to engine X

$$\dot{Q}_{1x} - \dot{Q}_{2x} = W_x$$

$$\therefore \dot{Q}_{2x} = \dot{Q}_{1x} - W_x = 780 \text{ kJ}$$

As the body at 600°C maintains its temperature steady, heat transfer to it must be equal to heat transfer from it. Therefore,

$$\dot{Q}_{2x} = \dot{Q}_{1y} = 780 \text{ kJ}$$

Applying the first law to engine Y.

$$\dot{Q}_{1y} - \dot{Q}_{2y} = W_y \quad \therefore W_y = (780 - 400) \text{ kJ} = \frac{380 \text{ kJ}}{\text{Ans}}$$

$$\eta_x = \frac{W_y}{\dot{Q}_{1y}} = \frac{380}{780} = 0.487 \quad (\text{Ans})$$

4) $(COP)_{HP} = 5, W = 35 \text{ kW, } \Delta t|_{\substack{\text{radiator} \\ \text{water}}} = 20^\circ\text{C}$

given

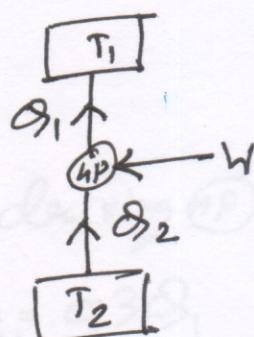
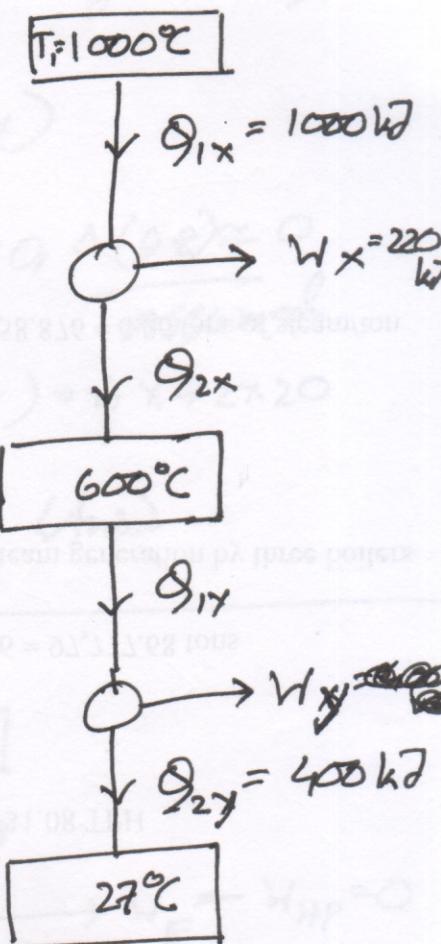
now $COP|_{HP} = \frac{\dot{Q}_1}{W} \Rightarrow \dot{Q}_1 = W \cdot (COP)_{HP}$

$$\therefore \dot{Q}_1 = 175 \text{ kW}$$

From 1st law: $\dot{Q}_1 - \dot{Q}_2 = W$

$$\dot{Q}_2 = \dot{Q}_1 - W = 140 \text{ kW}$$

↑ heat transfer to the heat pump(h.p.)



Applying the 1st law of TD to water flowing through the radiator

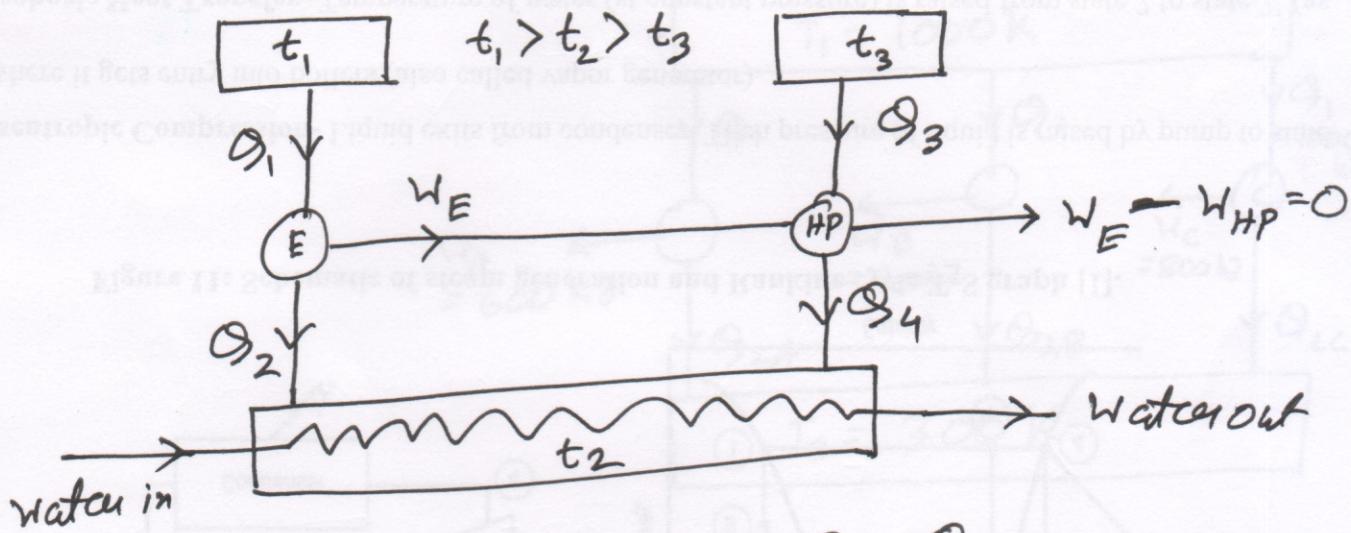
$$\dot{Q} - \dot{W} = \dot{m} \Delta \left(h + \frac{V^2}{2} + gz \right)$$

here, $\dot{Q} = \dot{Q}_1 = 175 \text{ kW}$, $\dot{W} = 0$, $\Delta \left(\frac{V^2}{2} \right) \approx 0$, $\Delta (gz) \approx 0$
Assumed

$$\therefore 175 - 0 = \dot{m} \Delta h = \dot{m} c_{\text{water}} (\Delta T) = \dot{m} \times 42 \times 20$$

$$\therefore \dot{m} = \frac{175}{42 \times 20} = 2.083 \frac{\text{kg}}{\text{s}} \quad (\text{Ans})$$

5)



$$\text{Efficiency of the heat engine} = \frac{\dot{Q}_1 - \dot{Q}_2}{\dot{Q}_1}$$

$$= 1 - \frac{\dot{Q}_2}{\dot{Q}_1} = 0.3$$

~~0.333~~

$$\therefore \dot{Q}_2 / \dot{Q}_1 = 0.7$$

Here, $W_E = W_{HP} \leftarrow$ because the (E) is driving (HP)

$$COP_{hp} = \eta = \frac{\dot{Q}_4}{W_{hp}} \quad \text{where, } W_{hp} = W_E = \eta \times \dot{Q}_1 = 0.3 \dot{Q}_1$$

$$\therefore \frac{\dot{Q}_4}{W_{hp}} = \frac{\dot{Q}_4}{0.3 \dot{Q}_1} = 4 \quad \therefore \frac{\dot{Q}_4}{\dot{Q}_1} = 0.3 \times 4 = 1.2$$

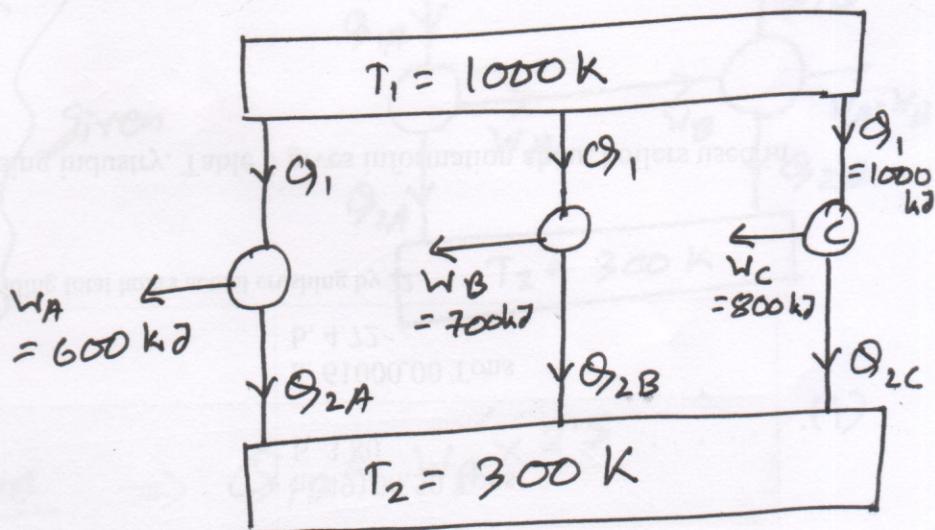
Total heat transfer to the radiator water = $Q_1 + Q_2$

Heat transfer to the engine = Q_1 ,

\therefore Heat transfer to radiator water for every kW added to the heat engine = $(Q_2 + Q_4)/Q_1$,

$$\begin{aligned} &= \frac{Q_2}{Q_1} + \frac{Q_4}{Q_1} \\ &= (0.7 + 1.2) \text{ kW} \\ &= 1.9 \text{ kW} \quad \langle \text{Ans} \rangle \end{aligned}$$

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Now, no engine operating between the two same temperature can not be more efficient a reversible engine.

$$\text{Here, } \eta_{\max} = \eta_{\text{rev}} = 1 - \frac{Q_{2\text{rev}}}{Q_{1\text{rev}}} = 1 - \frac{T_2}{T_1} = 1 - \frac{300}{1000} = 0.7$$

Engine A: $Q_1 = 1000\text{ kW}$, $W_A = 600\text{ kW}$

$$\therefore \eta_A = \frac{W_A}{Q_1} = 0.6$$

$\eta_A < \eta_{\max}$ — engine A: possible & irreversible

$$\text{Engine B: } w_B = 700 \text{ kJ} \quad \therefore \eta_B = \frac{w_B}{Q_1} = 0.7$$

$\eta_B = \eta_{\max}$ — engine B is ~~possible~~ reversible

$$\text{Engine C: } w_C = 800 \text{ kJ} \quad \therefore \eta_C = \frac{w_C}{Q_1} = 0.8$$

$\eta_C > \eta_{\max}$ — engine C is impossible

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$$T_1 = 273 + 177 = 450 \text{ K}$$

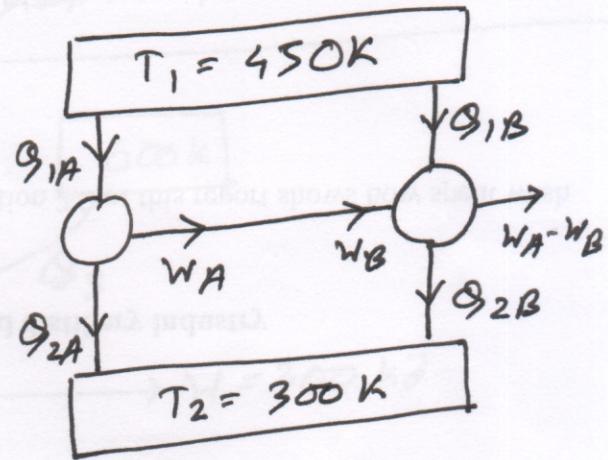
$$T_2 = 273 + 27 = 300 \text{ K}$$

$$Q_{2A} = 200 \text{ kJ}$$

$$Q_{2B} = 50 \text{ kJ}$$

$$COP_{HP, B} = 2.5$$

Given



$$COP_{HP, B} = 2.5 = \frac{Q_{1B}}{W_B} \Rightarrow Q_{1B} = W_B \times 2.5 \dots \dots \dots (i)$$

where W_B is the work required by the reversible heat engine B.

Applying the 1st law of TD engine Br —

$$Q_{1B} = W_B + Q_{2B} \dots \dots \dots (ii)$$

Eliminating Q_{1B} between (i) and (ii), and putting the value for Q_{2B}

$$W_B \times 2.5 = W_B + 50$$

$$\therefore W_B = 33.3 \text{ kJ}$$

From data given, $W_A = W_B + W_{output} = 33.3 + 20 = 53.3 \text{ kJ}$
 Applying the 1st law of TD to engine A,

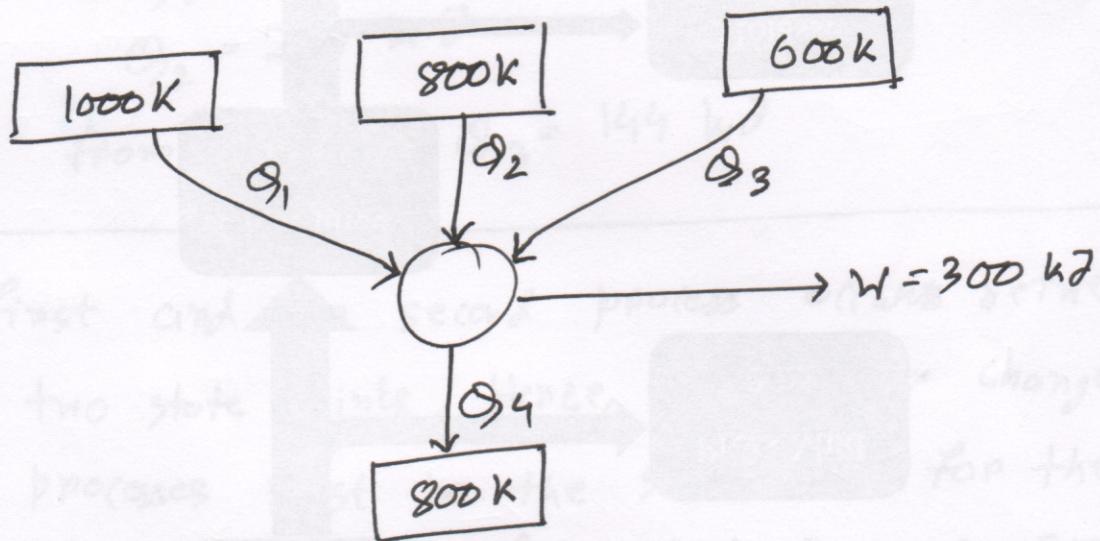
$$\dot{Q}_{1A} = \dot{Q}_{2A} + w_A = 200 + 53.3 = 253.3 \text{ kJ}$$

$$\eta_A = \frac{w_A}{\dot{Q}_{1A}} = \frac{53.3}{253.3} = 0.21$$

$$\eta_{max} = 1 - \frac{T_2}{T_1} = 1 - \frac{300}{450} = 0.33$$

$\therefore \eta_A < \eta_{max}$ \therefore engine A is ~~is~~ irreversible

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From 1st law of TD for the given engine, —

$$\dot{Q}_1 + \dot{Q}_2 + \dot{Q}_3 - \dot{Q}_4 = w$$

From given data: $\dot{Q}_3 = 2\dot{Q}_1$; $\dot{Q}_4 = 200 \text{ kJ}$; $w = 300 \text{ kJ}$

$$\therefore \dot{Q}_1 + \dot{Q}_2 + 2\dot{Q}_1 - 200 = 300$$

$$\therefore 3\dot{Q}_1 + \dot{Q}_2 = 500 \dots \dots \dots \dots \dots \dots \quad (i)$$

Also, as the cycle is reversible, from the Clausius inequality the cyclic integral of $\frac{\delta Q}{T} = 0$.
 Taking

Taking the directions of heat transfer with respect to the engine which is undergoing a cycle,

$$\oint \frac{dq}{T} = \frac{Q_1}{1000} + \frac{Q_2}{800} + \frac{Q_3}{600} - \frac{Q_4}{300} = 0$$

$$\therefore \cancel{\frac{Q_4}{1000}} + \frac{Q_1}{1000} + \frac{Q_2}{800} + \frac{2Q_3}{600} - \frac{2Q_4}{300} = 0 \dots\dots (ii)$$

Solving for Q_1 and Q_2 from (i) & (ii)

$$Q_1 = 72 \text{ kJ}$$

$$Q_2 = 284 \text{ kJ}$$

$$\text{from these } \Rightarrow Q_3 = 144 \text{ kJ}$$

9) The first and the second process occurs between the same two state points. Hence, the entropy change in both processes must be the same but for their direction. The process from 1 to 2 is irreversible because of stirring work. The process from 2 to 1 is given to be reversible and hence entropy change can be evaluated along this path.

$$TdS = dh - Vdp$$

$$\therefore dS = \frac{dh}{T} - \left(\frac{V}{T}\right)dp$$

For the process 2-1, p is constant and therefore

$$dS = \frac{dh}{T}$$

For the substance, C_p is a constant and $\therefore dh = C_p dT$

$$\therefore ds = C_p \frac{dT}{T}$$

$$\text{or, } \Delta S = S_2 - S_1 = \int C_p \frac{dT}{T} = C_p \ln\left(\frac{T_2}{T_1}\right)$$

Here, $T_1 = 30 + 273 = \cancel{300} 303 \text{ K}$

and $T_2 = 80 + 273 = 353 \text{ K}$

$$\therefore S_2 - S_1 = 0.6 \ln\left(\frac{303}{353}\right) = -0.0916 \frac{\text{kJ}}{\text{kg K}}$$

Now, entropy increase in the 1st pros. $= S_2 - S_1$

$$= -(S_1 - S_2)$$

$$= 0.0916 \frac{\text{kJ}}{\text{kg K}}$$

$$\therefore S_2 - S_1 = m(S_2 - S_1) = 2.5 \times 0.0916 \frac{\text{kJ}}{\text{K}} = 0.229 \frac{\text{kJ}}{\text{K}}$$

10) (a) For a pure substance; $- ds = \frac{dh}{T} - \left(\frac{V}{T}\right) dp$

Heating of water in a open vessel is a constant pressure process as the pressure is steady at the atmospheric value and hence in the equation: $dp = 0$

$$ds = \frac{dh}{T} = C \frac{dT}{T}$$

$$\therefore \Delta S = S_2 - S_1 = C \ln\left(\frac{T_2}{T_1}\right)$$

$$= 4.2 \ln\left(\frac{273+100}{273+30}\right)$$

$$= 0.873 \frac{\text{kJ}}{\text{kg K}}$$

Heat transfer, Q , to water, by the 1st law TD ~~cons~~ is given by: $Q = \Delta U + W$

Where the work $W=0$ as the process is also at constant volume (water is an incompressible substance) and ~~$\Delta U = 0$~~ $\Delta U = c(T_2 - T_1)$

$$\therefore Q = 4.2(100 - 30) = 294 \frac{\text{kJ}}{\text{kg}}$$

This heat is transferred from the reservoir to water. Heat transfer from the reservoir is internally reversible as its temperature remains the same.

Hence,

$$\Delta S_{\text{reservoir}} = \int \frac{\delta S_{\text{rev}}}{T} = \frac{Q}{T} = \frac{-294}{(273+100)} \frac{\text{kJ}}{\text{kg K}}$$

$$= -0.788 \frac{\text{kJ}}{\text{kg K}}$$

Total water

$$\Delta S_{\text{universe}} = \Delta S_{\text{water}} + \Delta S_{\text{reservoir}}$$

$$= (0.873 + (-0.788)) \frac{\text{kJ}}{(\text{kg water}) \text{K}}$$

$$= 0.085 \frac{\text{kJ}}{(\text{kg water}) \text{K}}$$

(b) water is heated first from 30°C to 65°C by heat transfer from the reservoir at 65°C and then from 65°C to 100°C by heat transfer from the ~~reservoir~~ reservoir at 100°C . Following the same sequence of calculation as in (a) for the first stage of

$$\Delta S_{\text{water},1} = c \ln \left(\frac{T_2}{T_1} \right) = 4.2 \ln \left(\frac{273+65}{273+30} \right) \frac{\text{kJ}}{\text{kg K}}$$

$$= 0.459 \frac{\text{kJ}}{\text{kg K}}$$

Heat transfer to water = $\dot{Q}_1 = C(t_2 - t_1)$

$$= 4.2 (65 - 30) \frac{\text{kJ}}{\text{kg}}$$

$$= 147 \frac{\text{kJ}}{\text{kg}}$$

$$\Delta S_{\text{pес}, I} = \int \frac{\delta \dot{Q}_{\text{pес}}}{T} = \frac{\dot{Q}}{T} = \frac{-147}{273+65} \frac{\text{kJ}}{(\text{kg water}) \text{K}}$$
$$= -0.435 \frac{\text{kJ}}{(\text{kg water}) \text{K}}$$

Similarly, $\Delta S_{\text{water}, II} = C \ln \left(\frac{T_2}{T_1} \right) = 4.2 \ln \frac{273+100}{273+65} \frac{\text{kJ}}{\text{kg water}}$

$$= 0.414 \frac{\text{kJ}}{\text{kg K}}$$

Heat transfer to water = $\dot{Q}_{II} = C(t_2 - t_1) = 4.2(1000 - 65) \frac{\text{kJ}}{\text{kg}}$

$$= 147 \frac{\text{kJ}}{\text{kg}}$$

$$\Delta S_{\text{pес}, II} = \int \frac{\delta \dot{Q}_{\text{pес}}}{T} = \frac{-147}{273+100} = -0.394 \frac{\text{kJ}}{(\text{kg water}) \text{K}}$$

$$\Delta S_{\text{universe}} = \Delta S_{\text{water}, I} + \Delta S_{\text{pесervoir}, I} + \Delta S_{\text{water}, II}$$

$$+ \Delta S_{\text{pесervoir}, II}$$

$$= [0.459 + (-0.435) + 0.414 + (-0.394)] \frac{\text{kJ}}{(\text{kg water})}$$

$$= 0.044 \frac{\text{kJ}}{(\text{kg water}) \text{K}}$$