#### **CHAPTER**

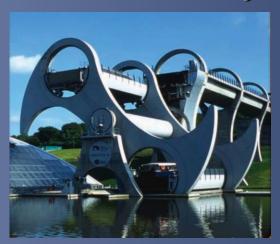
# 3

### VECTOR MECHANICS FOR ENGINEERS:

# **STATICS**

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Lecture Notes:
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Texas Tech University



Rigid Bodies: Equivalent Systems of Forces



#### Ninth Edition

# Vector Mechanics for Engineers: Statics

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#### Introduction

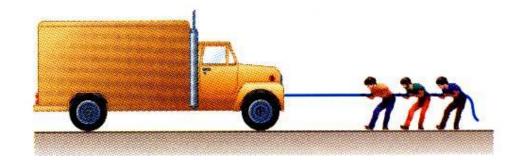
- Treatment of a body as a single particle is not always possible. In general, the size of the body and the specific points of application of the forces must be considered.
- Most bodies in elementary mechanics are assumed to be rigid, i.e., the actual deformations are small and do not affect the conditions of equilibrium or motion of the body.
- Current chapter describes the effect of forces exerted on a rigid body and how to replace a given system of forces with a simpler equivalent system.
  - moment of a force about a point
  - moment of a force about an axis
  - moment due to a couple
- Any system of forces acting on a rigid body can be replaced by an equivalent system consisting of one force acting at a given point and one couple.



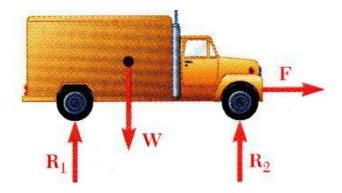


#### External and Internal Forces

- Forces acting on rigid bodies are divided into two groups:
  - External forces
  - Internal forces



 External forces are shown in a free-body diagram.



• If unopposed, each external force can impart a motion of translation or rotation, or both.

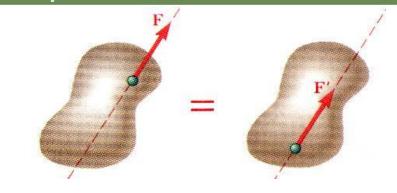




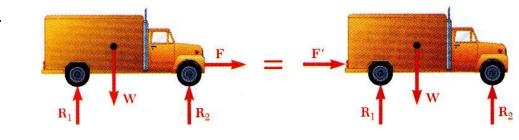
### Principle of Transmissibility: Equivalent Forces

• Principle of Transmissibility Conditions of equilibrium or motion are
not affected by transmitting a force
along its line of action.

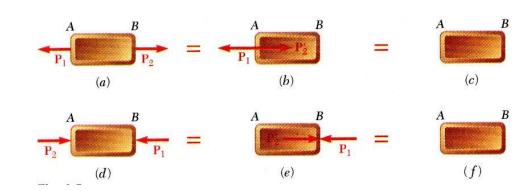
NOTE: **F** and **F**' are equivalent forces.



 Moving the point of application of the force F to the rear bumper does not affect the motion or the other forces acting on the truck.



• Principle of transmissibility may not always apply in determining internal forces and deformations.



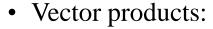


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# Vector Mechanics for Engineers: Statics

#### Vector Product of Two Vectors

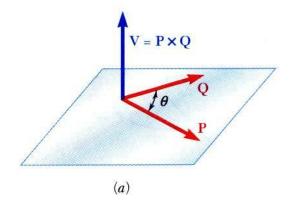
- Concept of the moment of a force about a point is more easily understood through applications of the *vector product* or *cross product*.
- Vector product of two vectors **P** and **Q** is defined as the vector **V** which satisfies the following conditions:
  - 1. Line of action of V is perpendicular to plane containing P and Q.
  - 2. Magnitude of *V* is  $V = PQ \sin q$
  - 3. Direction of *V* is obtained from the right-hand rule.

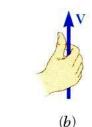


- are not commutative, 
$$Q \cdot P = -(P \cdot Q)$$

- are distributive, 
$$P'(Q_1 + Q_2) = P'Q_1 + P'Q_2$$

- are not associative,  $(P \ Q) \ S \ P \ (Q \ S)$ 









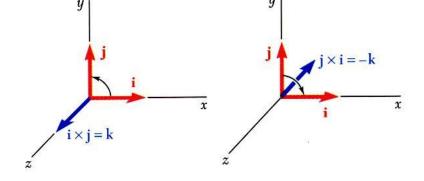
### Vector Products: Rectangular Components

• Vector products of Cartesian unit vectors,

$$\vec{i} \times \vec{i} = 0 \qquad \vec{j} \times \vec{i} = -\vec{k} \qquad \vec{k} \times \vec{i} = \vec{j}$$

$$\vec{i} \times \vec{j} = \vec{k} \qquad \vec{j} \times \vec{j} = 0 \qquad \vec{k} \times \vec{j} = -\vec{i}$$

$$\vec{i} \times \vec{k} = -\vec{j} \qquad \vec{j} \times \vec{k} = \vec{i} \qquad \vec{k} \times \vec{k} = 0$$



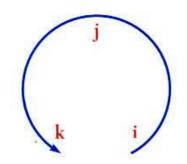
 Vector products in terms of rectangular coordinates

$$\vec{V} = (P_x \vec{i} + P_y \vec{j} + P_z \vec{k}) \times (Q_x \vec{i} + Q_y \vec{j} + Q_z \vec{k})$$

$$= (P_y Q_z - P_z Q_y) + (P_z Q_x - P_x Q_z) \vec{j}$$

$$+ (P_x Q_y - P_y Q_x) \vec{k}$$

$$= \begin{vmatrix} \vec{i} & \vec{j} & \vec{k} \\ P_x & P_y & P_z \\ Q_x & Q_y & Q_z \end{vmatrix}$$





#### Moment of a Force About a Point

- A force vector is defined by its magnitude and direction. Its effect on the rigid body also depends on it point of application.
- The *moment* of **F** about O is defined as

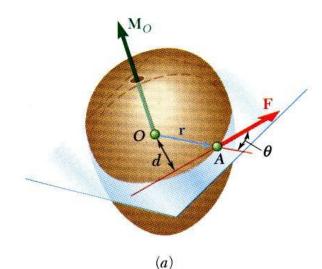
$$M_{O} = r \cdot F$$

- The moment vector  $M_0$  is perpendicular to the plane containing O and the force F.
- Magnitude of  $M_o$  measures the tendency of the force to cause rotation of the body about an axis along  $M_o$ .

$$M_o = rF \sin \theta = Fd$$

The sense of the moment may be determined by the right-hand rule.

• Any force F' that has the same magnitude and direction as F, is *equivalent* if it also has the same line of action and therefore, produces the same moment.





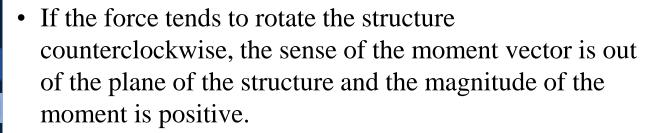




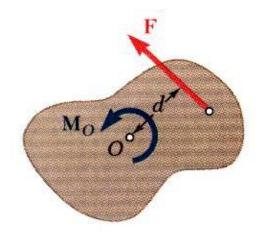


#### Moment of a Force About a Point

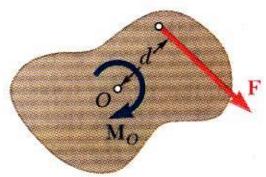
- *Two-dimensional structures* have length and breadth but negligible depth and are subjected to forces contained in the plane of the structure.
- The plane of the structure contains the point O and the force F.  $M_O$ , the moment of the force about O is perpendicular to the plane.



• If the force tends to rotate the structure clockwise, the sense of the moment vector is into the plane of the structure and the magnitude of the moment is negative.



$$(a) M_O = + Fd$$



(b) 
$$M_O = -Fd$$



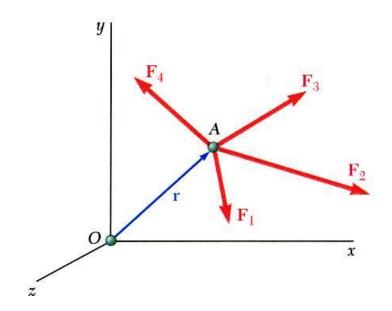


# Varignon's Theorem

• The moment about a give point *O* of the resultant of several concurrent forces is equal to the sum of the moments of the various moments about the same point *O*.

$$\vec{r} \times (\vec{F}_1 + \vec{F}_2 + \cdots) = \vec{r} \times \vec{F}_1 + \vec{r} \times \vec{F}_2 + \cdots$$

• Varigon's Theorem makes it possible to replace the direct determination of the moment of a force *F* by the moments of two or more component forces of *F*.







#### Rectangular Components of the Moment of a Force

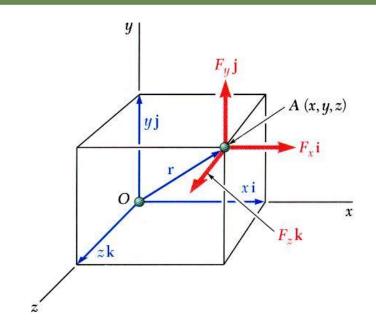
The moment of F about O,

$$\begin{split} \vec{M}_O &= \vec{r} \times \vec{F}, \quad \vec{r} = x \vec{i} + y \vec{j} + z \vec{k} \\ \vec{F} &= F_x \vec{i} + F_y \vec{j} + F_z \vec{k} \end{split}$$

$$\vec{M}_O = M_x \vec{i} + M_y \vec{j} + M_z \vec{k}$$

$$= \begin{vmatrix} \vec{i} & \vec{j} & \vec{k} \\ x & y & z \\ F_x & F_y & F_z \end{vmatrix}$$

$$= \left( yF_z - zF_y \right) \vec{i} + \left( zF_x - xF_z \right) \vec{j} + \left( xF_y - yF_x \right) \vec{k}$$







#### Rectangular Components of the Moment of a Force

The moment of F about B,

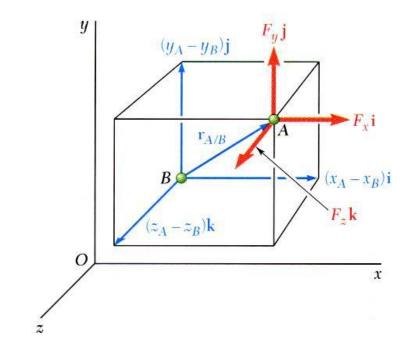
$$\vec{M}_B = \vec{r}_{A/B} \times \vec{F}$$

$$\vec{r}_{A/B} = \vec{r}_A - \vec{r}_B$$

$$= (x_A - x_B)\vec{i} + (y_A - y_B)\vec{j} + (z_A - z_B)\vec{k}$$

$$\vec{F} = F_x\vec{i} + F_y\vec{j} + F_z\vec{k}$$

$$\vec{M}_B = \begin{vmatrix} \vec{i} & \vec{j} & \vec{k} \\ (x_A - x_B) & (y_A - y_B) & (z_A - z_B) \\ F_x & F_y & F_z \end{vmatrix}$$







#### Rectangular Components of the Moment of a Force

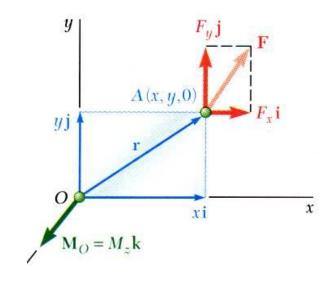
For two-dimensional structures,

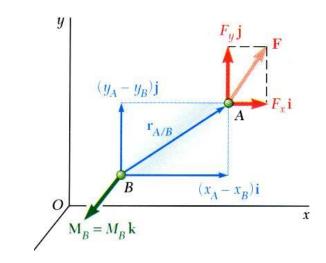
$$\vec{M}_O = (xF_y - yF_z)\vec{k}$$

$$M_O = M_Z$$

$$= xF_y - yF_z$$

$$\begin{split} \vec{M}_O &= \left[ (x_A - x_B) F_y - (y_A - y_B) F_z \right] \vec{k} \\ M_O &= M_Z \\ &= (x_A - x_B) F_y - (y_A - y_B) F_z \end{split}$$

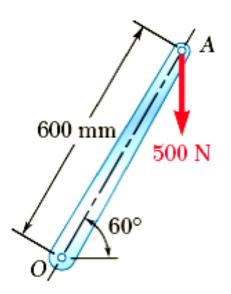








### Sample Problem 3.1



A 500-N vertical force is applied to the end of a lever which is attached to a shaft at *O*.

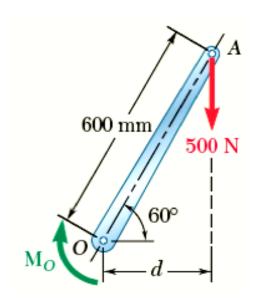
#### Determine:

- a) moment about O,
- b) horizontal force at A which creates the same moment,
- c) smallest force at A which produces the same moment,
- d) location for a 1200-N vertical force to produce the same moment,
- e) whether any of the forces from b, c, and d is equivalent to the original force.





### Sample Problem 3.1



#### **SOLUTION:**

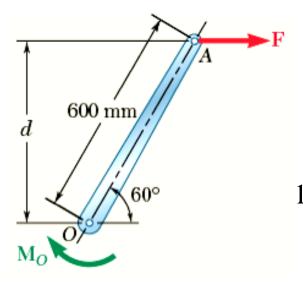
a) Moment about *O* is equal to the product of the force and the perpendicular distance between the line of action of the force and *O*. Since the force tends to rotate the lever clockwise, the moment vector is into the plane of the paper.

$$M_O = Fd$$
  
 $d = (600mm)\cos 60^\circ = 300mm = 0.3m$   
 $M_O = (500N)(0.3 \text{ m})$   $M_O = 150 \text{ N} \cdot \text{m}$ 





### Sample Problem 3.1



b) Horizontal force at A that produces the same moment,

$$d = (600 \text{ mm})\sin 60^\circ = 519.6 \text{ mm} = 0.5196 \text{ m}$$
 $M_O = Fd$ 
 $150 \text{ N} \cdot \text{m} = F(0.5196m)$ 

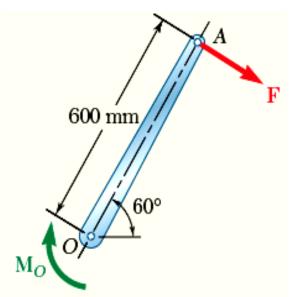
$$F = \frac{150 \text{ N.m}}{0.5196 \text{ m}}$$

F = 288.68 N





#### Sample Problem 3.1



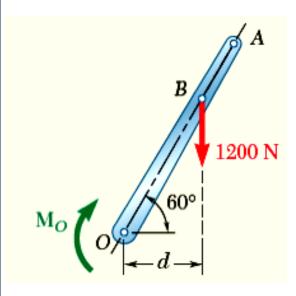
c) The smallest force *A* to produce the same moment occurs when the perpendicular distance is a maximum or when *F* is perpendicular to *OA*.

$$M_o = Fd$$
  
150 N.m =  $F(0.6 \text{ m})$   
 $F = \frac{150 \text{ N.m}}{0.6 \text{ m}}$   $F = 250 \text{ N}$ 





#### Sample Problem 3.1



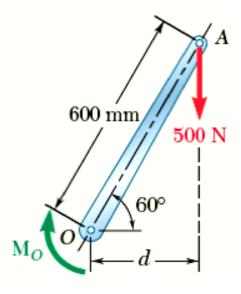
d) To determine the point of application of a 1200 N force to produce the same moment,

$$M_O = Fd$$
  
 $150 \text{ N.m} = (1200 \text{ N})d$   
 $d = \frac{150 \text{ N.m}}{1200 \text{ N}} = 125 \text{ mm}$   
 $OB \cos 60^\circ = 125 \text{ mm}$   $OB = 250 \text{ mm}$ 

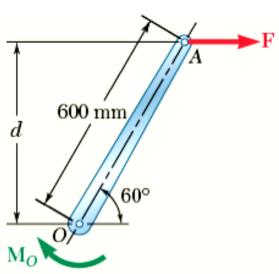


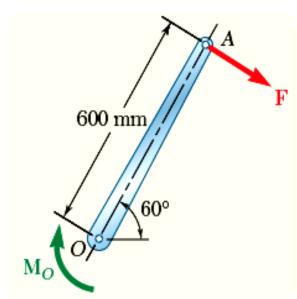


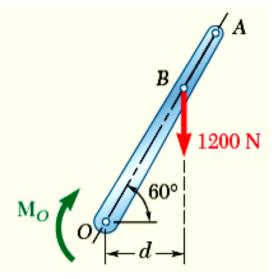
## Sample Problem 3.1



e) Although each of the forces in parts b), c), and d) produces the same moment as the 500-N force, none are of the same magnitude and sense, or on the same line of action. None of the forces is equivalent to the 500-N force.

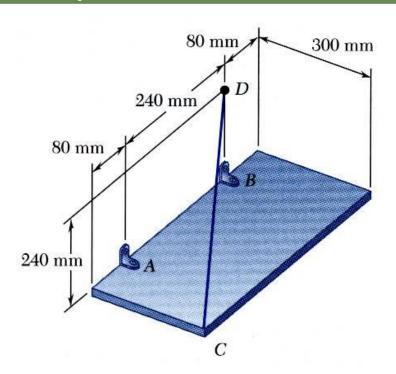








#### Sample Problem 3.4



The rectangular plate is supported by the brackets at *A* and *B* and by a wire *CD*. Knowing that the tension in the wire is 200 N, determine the moment about *A* of the force exerted by the wire at *C*.

#### **SOLUTION:**

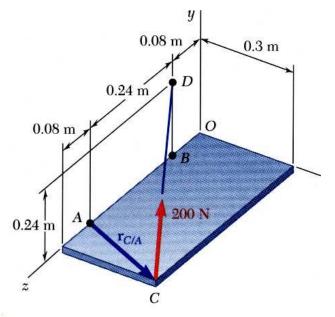
The moment  $M_A$  of the force F exerted by the wire is obtained by evaluating the vector product,

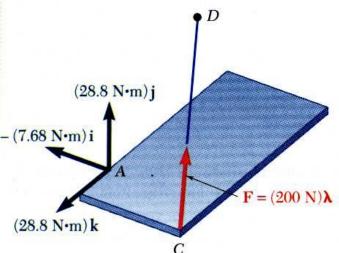
$$\vec{M}_A = \vec{r}_{C/A} \times \vec{F}$$





### Sample Problem 3.4





#### **SOLUTION:**

$$\vec{M}_A = \vec{r}_{C/A} \times \vec{F}$$

$$\vec{r}_{C/A} = \vec{r}_C - \vec{r}_A = (0.3 \text{ m})\vec{i} + (0.08 \text{ m})\vec{j}$$

$$\vec{F} = \vec{F} = (200 \text{ N}) \frac{\vec{r}_{C/D}}{r_{C/D}}$$

= 
$$(200 \text{ N})^{\frac{-(0.3 \text{ m})\vec{i} + (0.24 \text{ m})\vec{j} - (0.32 \text{ m})\vec{k}}{0.5 \text{ m}}$$

$$= -(120 \text{ N})\vec{i} + (96 \text{ N})\vec{j} - (128 \text{ N})\vec{k}$$

$$\vec{M}_A = \begin{vmatrix} \vec{i} & \vec{j} & \vec{k} \\ 0.3 & 0 & 0.08 \\ -120 & 96 & -128 \end{vmatrix}$$

$$\vec{M}_A = -(7.68 \text{ N} \cdot \text{m})\vec{i} + (28.8 \text{ N} \cdot \text{m})\vec{j} + (28.8 \text{ N} \cdot \text{m})\vec{k}$$

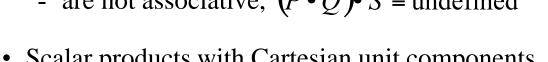


#### Scalar Product of Two Vectors

• The *scalar product* or *dot product* between two vectors  $\boldsymbol{P}$  and  $\boldsymbol{Q}$  is defined as

$$\vec{P} \cdot \vec{Q} = PQ \cos$$
 (scalar result)

- Scalar products:
  - are commutative,  $\vec{P} \cdot \vec{Q} = \vec{Q} \cdot \vec{P}$
  - are distributive,  $\vec{P} \cdot (\vec{Q}_1 + \vec{Q}_2) = \vec{P} \cdot \vec{Q}_1 + \vec{P} \cdot \vec{Q}_2$
  - are not associative,  $(\vec{P} \cdot \vec{Q}) \cdot \vec{S}$  = undefined



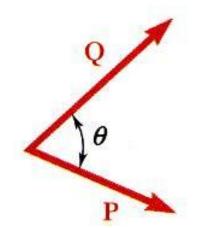
Scalar products with Cartesian unit components,

$$\vec{P} \cdot \vec{Q} = \left(P_x \vec{i} + P_y \vec{j} + P_z \vec{k}\right) \cdot \left(Q_x \vec{i} + Q_y \vec{j} + Q_z \vec{k}\right)$$

$$\vec{i} \cdot \vec{i} = 1$$
  $\vec{j} \cdot \vec{j} = 1$   $\vec{k} \cdot \vec{k} = 1$   $\vec{i} \cdot \vec{j} = 0$   $\vec{j} \cdot \vec{k} = 0$   $\vec{k} \cdot \vec{i} = 0$ 

$$\vec{P} \bullet \vec{Q} = P_x Q_x + P_y Q_y + P_z Q_z$$

$$\vec{P} \cdot \vec{P} = P_x^2 + P_y^2 + P_z^2 = P^2$$





#### Scalar Product of Two Vectors: Applications

• Angle between two vectors:

$$\vec{P} \cdot \vec{Q} = PQ\cos = P_x Q_x + P_y Q_y + P_z Q_z$$

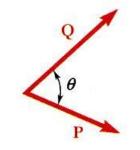
$$\cos = \frac{P_x Q_x + P_y Q_y + P_z Q_z}{PQ}$$

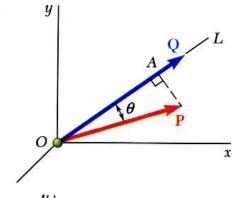
• Projection of a vector on a given axis:

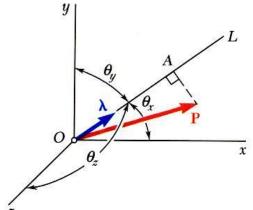
$$P_{OL} = P \cos = \text{projection of } P \text{ along } OL$$
  
 $\vec{P} \cdot \vec{Q} = PQ \cos$   
 $\frac{\vec{P} \cdot \vec{Q}}{O} = P \cos = P_{OL}$ 

$$P_{OL} = \vec{P} \bullet \vec{}$$

$$= P_x \cos_x + P_y \cos_y + P_z \cos_z$$





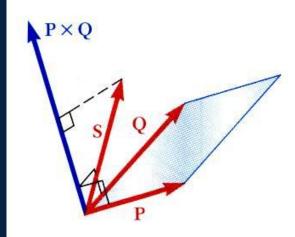




#### Ninth Edition

# Vector Mechanics for Engineers: Statics

#### Mixed Triple Product of Three Vectors



- Mixed triple product of three vectors,  $\vec{S} \cdot (\vec{P} \times \vec{Q}) = \text{scalar result}$
- The six mixed triple products formed from S, P, and
   Q have equal magnitudes but not the same sign,

$$\vec{S} \cdot (\vec{P} \times \vec{Q}) = \vec{P} \cdot (\vec{Q} \times \vec{S}) = \vec{Q} \cdot (\vec{S} \times \vec{P})$$

$$= -\vec{S} \cdot (\vec{Q} \times \vec{P}) = -\vec{P} \cdot (\vec{S} \times \vec{Q}) = -\vec{Q} \cdot (\vec{P} \times \vec{S})$$

Evaluating the mixed triple product,

$$\vec{S} \cdot (\vec{P} \times \vec{Q}) = S_x (P_y Q_z - P_z Q_y) + S_y (P_z Q_x - P_x Q_z) + S_z (P_x Q_y - P_y Q_x)$$

$$= \begin{vmatrix} S_x & S_y & S_z \\ P_x & P_y & P_z \\ Q_x & Q_y & Q_z \end{vmatrix}$$



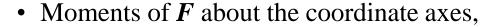
#### Moment of a Force About a Given Axis

• Moment  $M_o$  of a force F applied at the point A about a point O,

$$\vec{M}_O = \vec{r} \times \vec{F}$$

• Scalar moment  $M_{OL}$  about an axis OL is the projection of the moment vector  $M_O$  onto the axis,

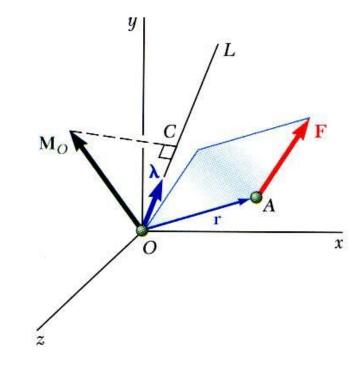
$$M_{OL} = \vec{\Phi} \cdot \vec{M}_O = \vec{\Phi} \cdot (\vec{r} \times \vec{F})$$



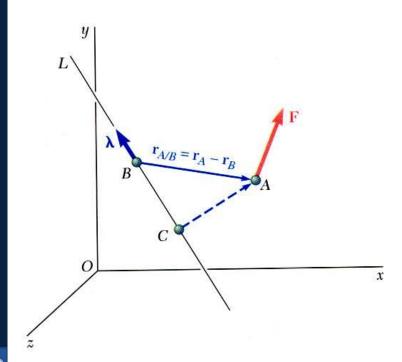
$$M_{x} = yF_{z} - zF_{y}$$

$$M_{y} = zF_{x} - xF_{z}$$

$$M_{z} = xF_{y} - yF_{x}$$



#### Moment of a Force About a Given Axis



• Moment of a force about an arbitrary axis,

$$M_{BL} = \vec{r} \cdot \vec{M}_{B}$$

$$= \vec{r}_{A/B} \times \vec{F}$$

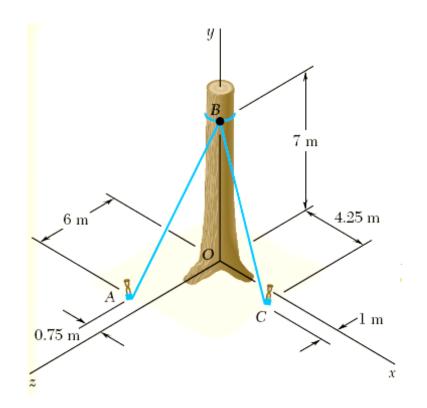
$$\vec{r}_{A/B} = \vec{r}_{A} - \vec{r}_{B}$$

• The result is independent of the point *B* along the given axis.





#### Sample Problem 3.5

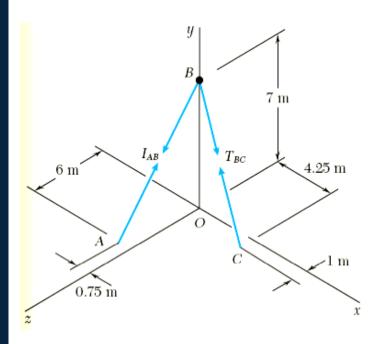


Before the trunk of a large tree is felled, cables *AB* and *BC* are attached as shown. Tensions in cables *AB* and *BC* are 555-N and 660-N, respectively.

Determine the moment about *O* of the resultant force exerted on the tree by the cables at *B*.



### Sample Problem 3.5



#### SOLUTION:

We have 
$$M_O = r_{B/o} \times F_B$$
  
where  $r_{B/o} = (7 \text{ m})\mathbf{j}$   
 $F_B = T_{AB} + T_{BC}$   
 $T_{AB} = \lambda_{BA} T_{AB}$   

$$= \frac{-(0.75m)\mathbf{i} - (7m)\mathbf{j} + (6m)\mathbf{k}}{\sqrt{(.75)^2 + (7)^2 + (6)^2m}} (555 \text{ N})$$
 $T_{BC} = \lambda_{BC} T_{BC}$   

$$= \frac{(4.25m)\mathbf{i} - (7m)\mathbf{j} + (1m)\mathbf{k}}{\sqrt{(4.25)^2 + (7)^2 + (1)^2m}} (660 \text{ N})$$

$$F_B = [-(45.00 \text{ N})\mathbf{i} - (420.0 \text{ N})\mathbf{j} + (360.0 \text{ N})\mathbf{k}]$$

$$+[(340.0 \text{ N})\mathbf{i} - (560.0 \text{ N})\mathbf{j} + (80.00 \text{ N})\mathbf{k}]$$

$$= (295.0 \text{ N})\mathbf{i} - (980.0 \text{ N})\mathbf{j} + (440.0 \text{ N})\mathbf{k}$$
and  $M_O = \begin{vmatrix} \mathbf{i} & \mathbf{j} & \mathbf{k} \\ 0 & 7 & 0 \\ 295 & 980 & 440 \end{vmatrix} \text{N} \cdot \text{m}$ 

$$= (3080 \text{ N} \cdot \text{m})\mathbf{i} - (2070 \text{ N} \cdot \text{m})\mathbf{k} \text{ or}$$

$$M_O = (3080 \text{ N} \cdot \text{m})\mathbf{i} - (2070 \text{ N} \cdot \text{m})\mathbf{k}$$



# Moment of a Couple

- Two forces **F** and **-F** having the same magnitude, parallel lines of action, and opposite sense are said to form a *couple*.
- Moment of the couple,

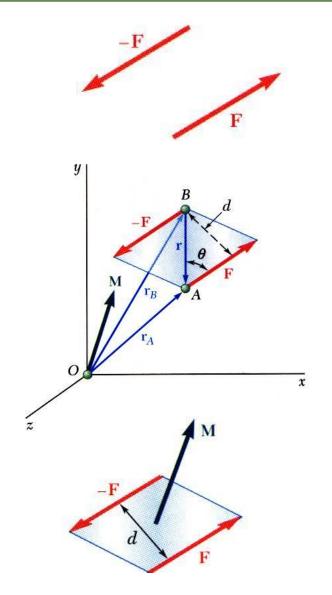
$$\vec{M} = \vec{r}_A \times \vec{F} + \vec{r}_B \times (-\vec{F})$$

$$= (\vec{r}_A - \vec{r}_B) \times \vec{F}$$

$$= \vec{r} \times \vec{F}$$

$$M = rF \sin \theta = Fd$$

• The moment vector of the couple is independent of the choice of the origin of the coordinate axes, i.e., it is a *free vector* that can be applied at any point with the same effect.





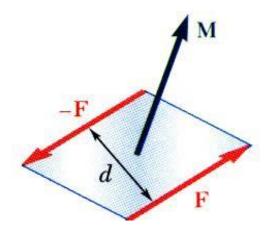


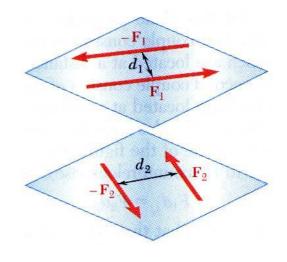
# Moment of a Couple

Two couples will have equal moments if

• 
$$F_1d_1 = F_2d_2$$

- the two couples lie in parallel planes, and
- the two couples have the same sense or the tendency to cause rotation in the same direction.









### Addition of Couples

• Consider two intersecting planes  $P_1$  and  $P_2$  with each containing a couple

$$\vec{M}_1 = \vec{r} \times \vec{F}_1$$
 in plane  $P_1$   
 $\vec{M}_2 = \vec{r} \times \vec{F}_2$  in plane  $P_2$ 

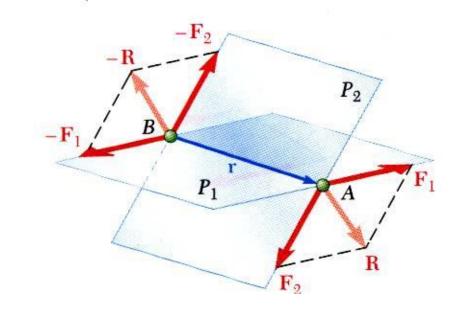
• Resultants of the vectors also form a couple

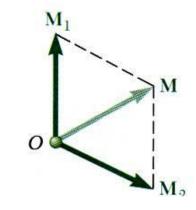
$$\vec{M} = \vec{r} \times \vec{R} = \vec{r} \times (\vec{F}_1 + \vec{F}_2)$$



$$\vec{M} = \vec{r} \times \vec{F}_1 + \vec{r} \times \vec{F}_2$$
$$= \vec{M}_1 + \vec{M}_2$$

• Sum of two couples is also a couple that is equal to the vector sum of the two couples

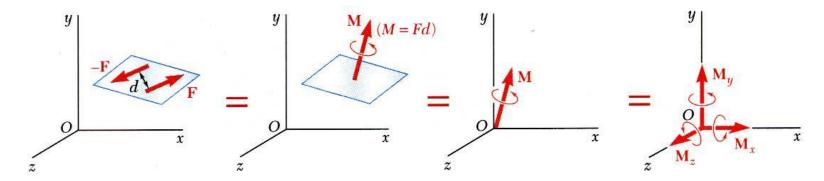








# Couples Can Be Represented by Vectors

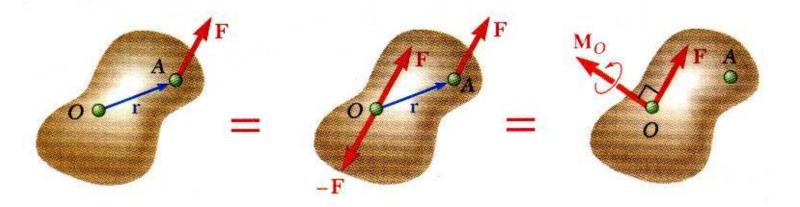


- A couple can be represented by a vector with magnitude and direction equal to the moment of the couple.
- Couple vectors obey the law of addition of vectors.
- Couple vectors are free vectors, i.e., the point of application is not significant.
- Couple vectors may be resolved into component vectors.





#### Resolution of a Force Into a Force at O and a Couple



- Force vector **F** can not be simply moved to *O* without modifying its action on the body.
- Attaching equal and opposite force vectors at *O* produces no net effect on the body.
- The three forces may be replaced by an equivalent force vector and couple vector, i.e, a *force-couple system*.

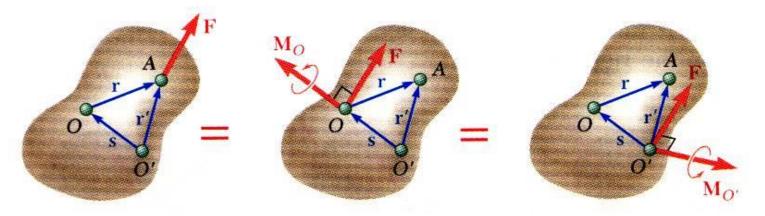




#### Ninth Edition

# Vector Mechanics for Engineers: Statics

#### Resolution of a Force Into a Force at O and a Couple



• Moving F from A to a different point O' requires the addition of a different couple vector  $M_{O'}$ 

$$\vec{M}_{O'} = \vec{r}' \times \vec{F}$$

• The moments of **F** about O and O' are related,

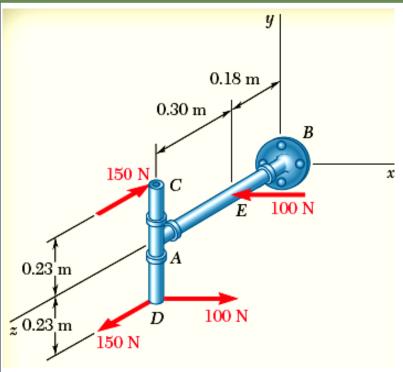
$$\vec{M}_{O'} = \vec{r}' \times \vec{F} = (\vec{r} + \vec{s}) \times \vec{F} = \vec{r} \times \vec{F} + \vec{s} \times \vec{F}$$
$$= \vec{M}_{O} + \vec{s} \times \vec{F}$$

• Moving the force-couple system from O to O' requires the addition of the moment of the force at O about O'.





### Sample Problem 3.6



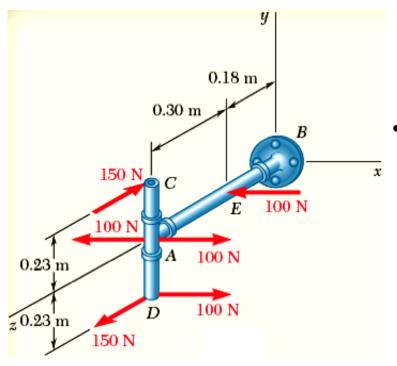
Determine the components of the single couple equivalent to the couples shown.

#### **SOLUTION:**

- Attach equal and opposite 100-N forces in the  $\pm x$  direction at A, thereby producing 3 couples for which the moment components are easily computed.
- Alternatively, compute the sum of the moments of the four forces about an arbitrary single point. The point *D* is a good choice as only two of the forces will produce non-zero moment contributions..



#### Sample Problem 3.6



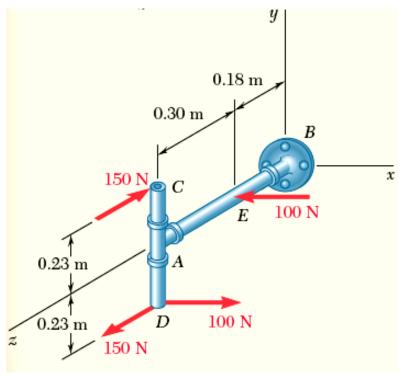
 The three couples may be represented by three couple vectors,

$$M_x = -(150 \text{ N})(0.45 \text{ m}) = -67.5 \text{ N} \cdot \text{m}$$
  
 $M_y = +(100 \text{ N})(0.30 \text{ m}) = +30 \text{ N} \cdot \text{m}$   
 $M_z = +(100 \text{ N})(0.23 \text{ m}) = +22.5 \text{ N} \cdot \text{m}$ 

$$\vec{M} = -(67.5 \text{ N.m})\vec{i} + (30 \text{ N.m})\vec{j} + (22.5 \text{ N.m})\vec{k}$$



### Sample Problem 3.6



- Alternatively, compute the sum of the moments of the four forces about *D*.
- Only the forces at *C* and *E* contribute to the moment about *D*.

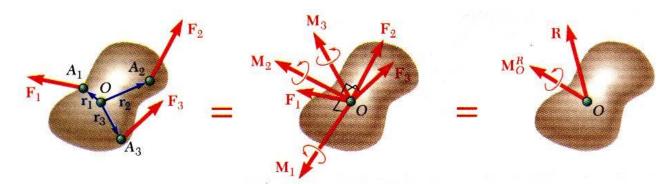
$$\vec{M} = \vec{M}_D = (0.45 \text{ m})\vec{j} \times (-150 \text{ N})\vec{k}$$
  
+  $[(0.23 \text{ m})\vec{j} - (0.30 \text{ m})\vec{k}] \times (-100 \text{ N})\vec{i}$ 

$$\vec{M} = -(67.5 \cdot \text{m})\vec{i} + (30 \text{ N} \cdot \text{m})\vec{j} + (22.5 \text{ N} \cdot \text{m})\vec{k}$$





# System of Forces: Reduction to a Force and Couple



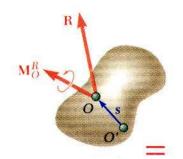
- A system of forces may be replaced by a collection of force-couple systems acting a given point *O*
- The force and couple vectors may be combined into a resultant force vector and a resultant couple vector,

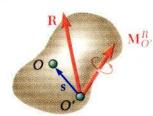
$$\vec{R} = \sum \vec{F}$$
  $\vec{M}_O^R = \sum (\vec{r} \times \vec{F})$ 

• The force-couple system at O may be moved to O' with the addition of the moment of **R** about O',

$$\vec{M}_{O'}^{R} = \vec{M}_{O}^{R} + \vec{s} \times \vec{R}$$

• Two systems of forces are equivalent if they can be reduced to the same force-couple system.

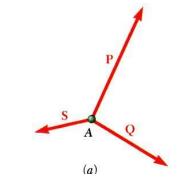


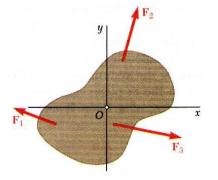


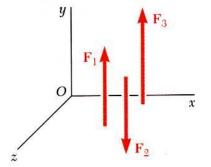


### Further Reduction of a System of Forces

- If the resultant force and couple at *O* are mutually perpendicular, they can be replaced by a single force acting along a new line of action.
- The resultant force-couple system for a system of forces will be mutually perpendicular if:
  - 1) the forces are concurrent,
  - 2) the forces are coplanar, or
  - 3) the forces are parallel.



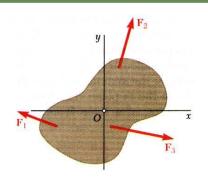




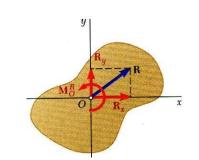


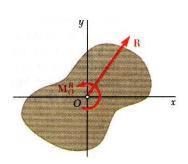


### Further Reduction of a System of Forces

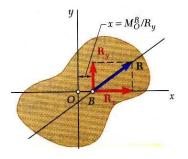


• System of coplanar forces is reduced to a force-couple system  $\vec{R}$  and  $\vec{M}_O^R$  that is mutually perpendicular.



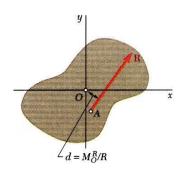


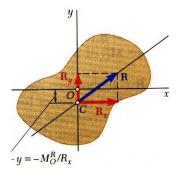
• System can be reduced to a single force by moving the line of action of  $\vec{R}$  until its moment about O becomes  $\vec{M}_{O}^{R}$ 



• In terms of rectangular coordinates,

$$xR_y - yR_x = M_O^R$$

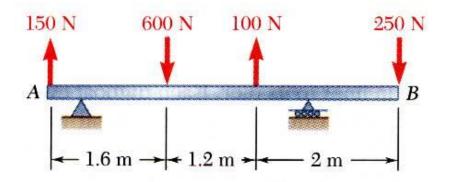








### Sample Problem 3.8



For the beam, reduce the system of forces shown to (a) an equivalent force-couple system at A, (b) an equivalent force couple system at B, and (c) a single force or resultant.

Note: Since the support reactions are not included, the given system will not maintain the beam in equilibrium.

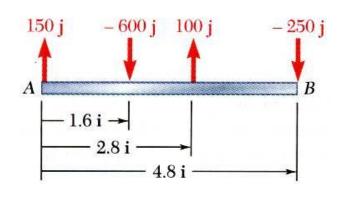
#### **SOLUTION:**

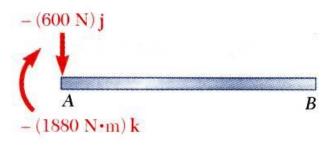
- a) Compute the resultant force for the forces shown and the resultant couple for the moments of the forces about *A*.
- b) Find an equivalent force-couple system at *B* based on the force-couple system at *A*.
- c) Determine the point of application for the resultant force such that its moment about *A* is equal to the resultant couple at *A*.





### Sample Problem 3.8





#### **SOLUTION:**

a) Compute the resultant force and the resultant couple at *A*.

$$\vec{R} = \sum \vec{F}$$
=  $(150 \text{ N})\vec{j} - (600 \text{ N})\vec{j} + (100 \text{ N})\vec{j} - (250 \text{ N})\vec{j}$ 

$$\vec{R} = -(600 \text{ N})\vec{j}$$

$$\vec{M}_{A}^{R} = \sum_{i} (\vec{r} \times \vec{F})$$

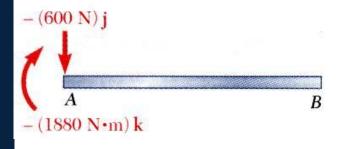
$$= (1.6\vec{i}) \times (-600\vec{j}) + (2.8\vec{i}) \times (100\vec{j})$$

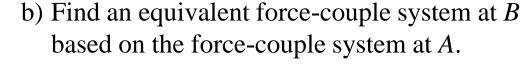
$$+ (4.8\vec{i}) \times (-250\vec{j})$$

$$\vec{M}_A^R = -(1880 \text{ N} \cdot \text{m})\vec{k}$$



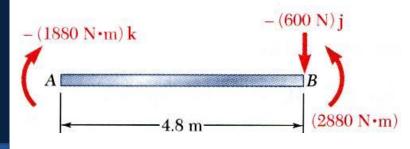
# Sample Problem 3.8





The force is unchanged by the movement of the force-couple system from *A* to *B*.

$$\vec{R} = -(600 \text{ N})\vec{j}$$



The couple at *B* is equal to the moment about *B* of the force-couple system found at A.

$$\vec{M}_{B}^{R} = \vec{M}_{A}^{R} + \vec{r}_{B/A} \times \vec{R}$$

$$= -(1880 \text{ N} \cdot \text{m})\vec{k} + (-4.8 \text{ m})\vec{i} \times (-600 \text{ N})\vec{j}$$

$$= -(1880 \text{ N} \cdot \text{m})\vec{k} + (2880 \text{ N} \cdot \text{m})\vec{k}$$

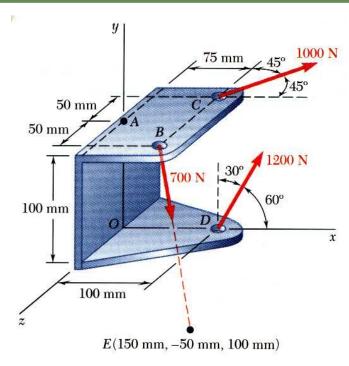
$$\vec{M}_{B}^{R} = +(1000 \text{ N} \cdot \text{m})\vec{k}$$

(1000 N·m) k





### Sample Problem 3.10



Three cables are attached to the bracket as shown. Replace the forces with an equivalent force-couple system at *A*.

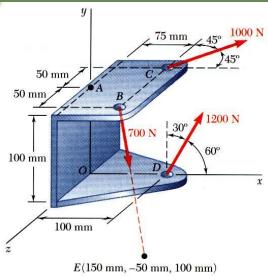
#### **SOLUTION:**

- Determine the relative position vectors for the points of application of the cable forces with respect to *A*.
- Resolve the forces into rectangular components.
- Compute the equivalent force,  $\vec{R} = \sum \vec{F}$
- Compute the equivalent couple,  $\vec{M}_{A}^{R} = \sum_{i} (\vec{r} \times \vec{F})$





### Sample Problem 3.10



#### **SOLUTION:**

• Determine the relative position vectors with respect to *A*.

$$\vec{r}_{B/A} = 0.075 \vec{i} + 0.050 \vec{k} \text{ (m)}$$

$$\vec{r}_{C/A} = 0.075 \vec{i} - 0.050 \vec{k} \text{ (m)}$$

$$\vec{r}_{D/A} = 0.100 \vec{i} - 0.100 \vec{j} \text{ (m)}$$

• Resolve the forces into rectangular components.

$$\vec{F}_B = (700 \text{ N})^{\vec{-}}$$

$$= \frac{\vec{r}_{E/B}}{r_{E/B}} = \frac{75\vec{i} - 150\vec{j} + 50\vec{k}}{175}$$

$$= 0.429\vec{i} - 0.857\vec{j} + 0.289\vec{k}$$

$$\vec{F}_B = 300\vec{i} - 600\vec{j} + 200\vec{k} \text{ (N)}$$

$$\vec{F}_C = (1000 \text{ N})(\cos 45 \vec{i} - \cos 45 \vec{j})$$
  
=  $707 \vec{i} - 707 \vec{j}$  (N)

$$\vec{F}_D = (1200 \text{ N})(\cos 60 \vec{i} + \cos 30 \vec{j})$$
  
=  $600 \vec{i} + 1039 \vec{j}$  (N)

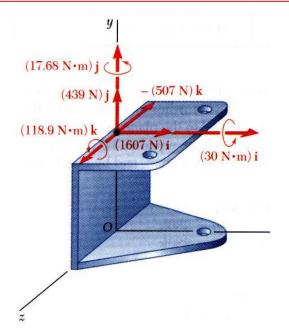


### Sample Problem 3.10

• Compute the equivalent force,

$$\vec{R} = \sum \vec{F}$$
=  $(300 + 707 + 600)\vec{i}$   
+  $(-600 + 1039)\vec{j}$   
+  $(200 - 707)\vec{k}$ 

$$\vec{R} = 1607\vec{i} + 439\vec{j} - 507\vec{k} \text{ (N)}$$



• Compute the equivalent couple,

$$\vec{M}_{A}^{R} = \sum (\vec{r} \times \vec{F})$$

$$\vec{r}_{B/A} \times \vec{F}_{B} = \begin{vmatrix} \vec{i} & \vec{j} & \vec{k} \\ 0.075 & 0 & 0.050 \\ 300 & -600 & 200 \end{vmatrix} = 30\vec{i} - 45\vec{k}$$

$$\vec{r}_{C/A} \times \vec{F}_c = \begin{vmatrix} \vec{i} & \vec{j} & \vec{k} \\ 0.075 & 0 & -0.050 \\ 707 & 0 & -707 \end{vmatrix} = 17.68\vec{j}$$

$$\vec{r}_{D/A} \times \vec{F}_D = \begin{vmatrix} \vec{i} & \vec{j} & \vec{k} \\ 0.100 & -0.100 & 0 \\ 600 & 1039 & 0 \end{vmatrix} = 163.9\vec{k}$$

$$\vec{M}_A^R = 30\vec{i} + 17.68\vec{j} + 118.9\vec{k}$$

