Question 1:

A 2-DOF (degree of freedom) mechanical system consists of two masses, $m_1 = 2 \,\mathrm{kg}$ and $m_2 = 1 \,\mathrm{kg}$, connected by springs with stiffness values $k_1 = 50 \,\mathrm{N/m}$, $k_2 = 100 \,\mathrm{N/m}$, and $k_3 = 50 \,\mathrm{N/m}$. The masses are arranged such that:

- Mass m_1 is connected to a wall with spring k_1 ,
- Mass m_1 and m_2 are connected by spring k_2 ,
- Mass m_2 is connected to a wall with spring k_3 .
- (a) Formulate the equations of motion for the system.
- (b) Use MATLAB to determine the natural frequencies and mode shapes of the system.

$$\int_{R_{1}} \frac{1}{x_{1}} = -k_{1}x_{1} + k_{2}(x_{2} - x_{1})$$

$$\int_{R_{1}} \frac{1}{x_{1}} = -(k_{1} + k_{2})x_{1} + k_{2}x_{2}$$

$$\int_{R_{1}} \frac{1}{x_{1}} = -(k_{1} + k_{2})x_{1} + k_{2}x_{2}$$

$$\int_{R_{1}} \frac{1}{x_{1}} = -k_{3}x_{1} - k_{2}x_{2}$$

$$\int_{R_{2}} \frac{1}{x_{2}} = -k_{3}x_{2} - k_{2}(x_{2} - x_{1})$$

$$\int_{R_{2}} \frac{1}{x_{2}} = -(k_{2} + k_{3})x_{2} + k_{2}x_{1}$$

$$\int_{R_{2}} \frac{1}{x_{2}} + (k_{2} + k_{3})x_{2} - k_{2}x_{1} = 0$$

$$\int_{R_{2}} \frac{1}{x_{2}} = -(k_{2} + k_{3})x_{2} - k_{2}x_{1} = 0$$

: Mass Matrix:

$$M = \begin{bmatrix} m_1 & 0 \\ 0 & m_2 \end{bmatrix}$$

Stiffners Matrix:

$$K = \begin{bmatrix} k_1 + k_2 & -k_2 \\ -k_2 & k_2 + k_3 \end{bmatrix}$$

Such that
$$M\ddot{x} = -Kx$$

where, $\ddot{x} = \begin{bmatrix} \ddot{x}_1 \\ \dot{x}_2 \end{bmatrix}$ and $x = \begin{bmatrix} x_1 \\ x_2 \end{bmatrix}$

Now, putting values of
$$\begin{cases} m_1 = 2kg & k_1 = 50 \text{ N/m} \\ m_2 = 1kg & k_2 = 100 \text{ N/m} \\ k_3 = 50 \text{ N/m} \end{cases}$$

i son of Hothon -

$$2\dot{x}_{1} = -150 x_{1} + 160 x_{2}$$

$$\dot{x}_{2} = -150 x_{2} + 150 x_{1}$$

CODE

```
%Given values
m1 = 2; %kg
m2 = 1; %kg
k1 = 50; %N/m
k2 = 100; %N/m
k3 = 50; %N/m
% Mass and stiffness matrices
M = [m1 \ 0; \ 0 \ m2];
K = [k1+k2 -k2;-k2 k2+k3];
% Solve the generalized eigenvalue problem
[phi, omega2] = eig(K, M);
% Natural frequencies (rad/s)
omega = sqrt(diag(omega2));
% Convert to Hz
frequencies_Hz = omega / (2*pi);
% Display results
disp('Natural frequencies in Hz:');
disp(frequencies_Hz);
disp('Mode shapes (normalized):');
disp(phi);
OUTPUT:
Natural frequencies in Hz:
  0.9068
  2.2084
Mode shapes (normalized):
  -0.6059 -0.3645
```

-0.5155 0.8569

- 1. For mode 1, both the masses move in the same direction, with m_1 moving faster than m_2 .
- 2. For mode 2, the masses move in the opposite direction.