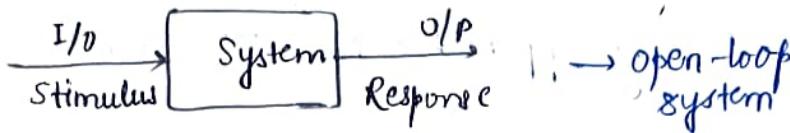


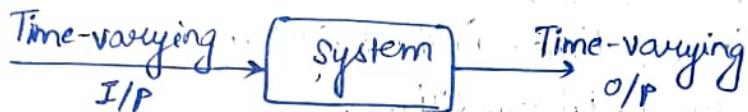
Feedback

System:



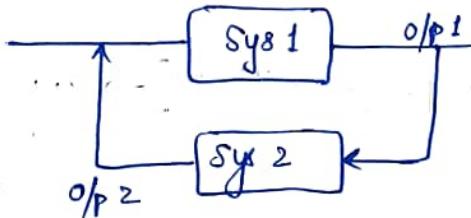
Youtube:
 → M. Gopal on Control Systems
 → C.S. RAM (IITM) control systems.

Dynamical system:



Feedback:

- ↳ Related to dynamical system
- ↳ Interconnection of two dynamical systems such that their outputs depend on each other.



→ circular argument
 → closed-loop system

Control System

- ↳ Classical control → Freq. domain analysis
- ↳ Modern Control
 - ↳ Time-domain / state-space analysis

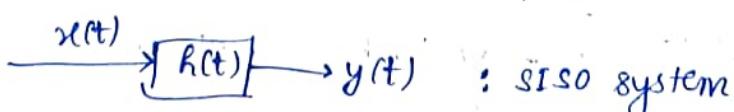
Books:

- Norman Nise
 - ↳ Control sys
- K. Ogata → Modern control Engg.
 - ↳ Digital control
- Dorf & Bishop
 - ↳ Control sys

Frequency-domain analysis:

- ↳ Transfer Function approach

→ T.F. : $\frac{\text{LT of O/P}}{\text{LT of I/P}}$, under the cond'n of zero initial state.



$$y(t) = x(t) * h(t) \Rightarrow Y(s) = X(s) \cdot H(s) \Rightarrow H(s) = \frac{Y(s)}{X(s)}$$

- T.F. approach (classical approach) deals with SISO system only.
 - ↳ For LTI system.
- Convolution is only applicable for LTI system.

State-Space Approach:

- ↳ Applicable to MIMO system also.
- ↳ Non-linear, time-variant system.
- ↳ Non-zero initial conditions.

Frequency Domain Analysis

- ↳ Routh-Hurwitz plot

- ↳ Bode plot

- ↳ Nyquist plot

All governed by

$$1 + G(s) H(s) = 0$$

characteristic eqn
for negative feedback system

14-08-2025

Feedback System

- ↳ Also known as closed loop system.

Control system performance matrix:

① Transient response

② Steady state error.

Sampling theorem:

$$f_s < 2f_m$$

↳ can be constructed.

$$1 + G(s) H(s) = 0 \rightarrow R.L., Bode plot, N.P.$$

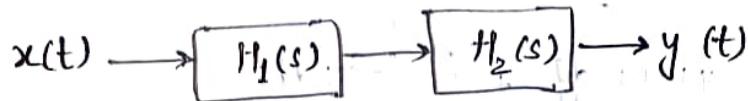
also called openloop gain.

$G(s)$: forward path gain.

Simplifying Blocks :

- ① cascade / series connection
- ② Parallel connection
- ③ Feedback connection

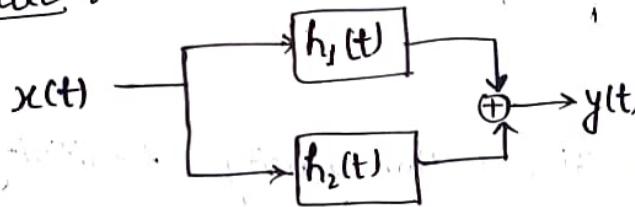
Series :



$$y(t) = x(t) * h_1(t) * h_2(t)$$

- In Laplace domain, we get into whether a system will decay or diverge (stability).
- In Fourier domain, we do not get such information.

Parallel :



$$y(t) = x(t) * h_1(t) + x(t) * h_2(t)$$

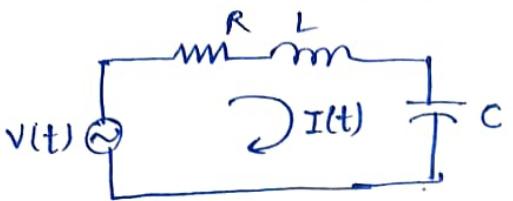
Feedback :



$$\text{T.F.} = \frac{G_1(s)}{1 + G_1(s) \cdot H(s)}$$

Some Terminologies Related to Systems

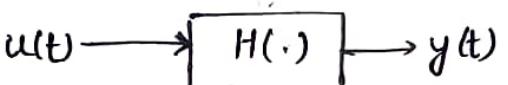
① Linear and Non-linear Systems:



$$V(t) = RI(t) + L \frac{dI}{dt} + \frac{1}{C} \int I(t) dt$$

Follows additivity \Rightarrow Linear system
and homogeneity

Diode \rightarrow Non-linear (exponential or parabolic)



$$y(t) = H(u(t))$$

Linearity = superposition = Homogeneity + Additivity

① Homogeneity / scaling:

$$u_1(t) \rightarrow [H(\cdot)] \rightarrow y_1(t)$$

$$u_2(t) \rightarrow [H(\cdot)] \rightarrow y_2(t)$$

$$c_1 u_1(t) \rightarrow [H(\cdot)] \rightarrow c_1 y_1(t), c_1 \in \mathbb{R}$$

② Additivity:

$$u_1(t) + u_2(t) \rightarrow [H(\cdot)] \rightarrow y_1(t) + y_2(t)$$

combining ① and ②,

$$c_1 u_1(t) + c_2 u_2(t) \rightarrow [H(\cdot)] \rightarrow c_1 y_1(t) + c_2 y_2(t).$$

$y = mx + c \rightarrow$ Affine
(Non-linear)

② Causal and Non-Causal Systems

Causal systems: Practical systems

↳ Output at any instant depends on past and present inputs.

$$y(t) = 2x(t) + 4x(t-2)$$

memory

Non-causal: Also depends on future inputs.

③ SISO and MIMO:

SISO: Single input and single output
 $u(t) \in \mathbb{R}$ $y(t) \in \mathbb{R}$

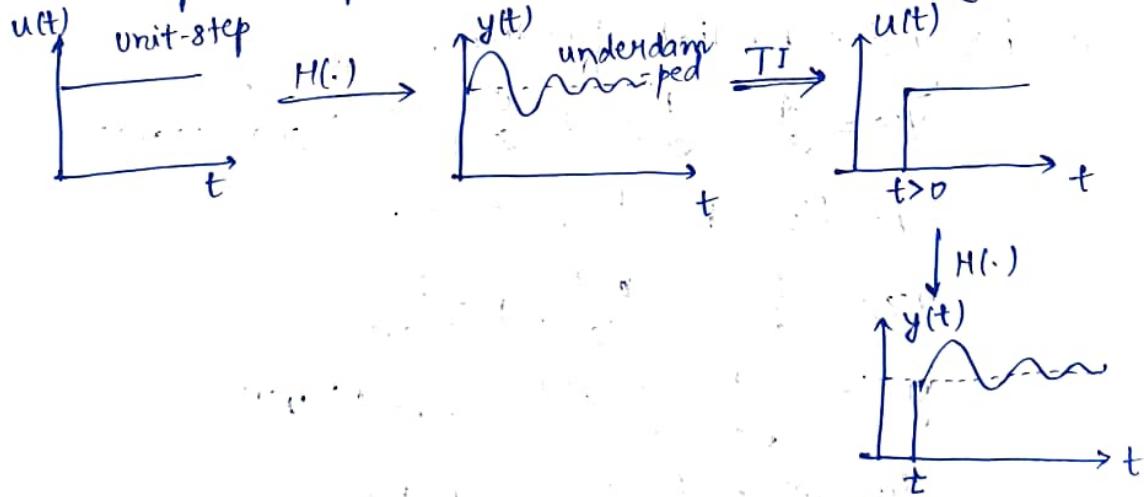
Federated Learning

Edge Computing

MIMO: Multiple input and multiple output.

④ Time Invariant and Time Varying:

TI: Input-output relationship never changes with time.



↳ Doesn't depend at which instant i/p is applied;
output pattern/shape remains the same.

BIBO Stability [Bounded Input, Bounded Output]

For any bounded i/p, $u(t)$, i.e., $|u(t)| \leq M < \infty$, $\forall t \geq 0$.

The system response, $y(t)$, is bounded,

i.e., $|y(t)| \leq N < \infty$, $\forall t \geq 0$.



→ Need for unit step function:

For analysis of sudden disturbances:

Eg. $\ddot{y}(t) + 5\dot{y}(t) + 6y(t) = u(t)$. Find the unit step response.

Taking LT both sides, $U(s) = 1/s$,

$$s^2 Y(s) + 5s Y(s) + 6Y(s) = U(s)$$

$$Y(s) = \frac{U(s)}{s^2 + 5s + 6}$$

$$H(s) = \frac{Y(s)}{U(s)} = \frac{1}{s^2 + 5s + 6} = \frac{1}{(s+2)(s+3)}$$

$$= \frac{1}{s+2} - \frac{1}{s+3} \rightarrow \text{Poles: } -2, -3$$

$$\therefore Y(s) = \frac{1}{s(s+2)} - \frac{1}{s(s+3)}$$

Use heavy-side partial fraction,

$$Y(s) = \frac{A}{s} + \frac{B}{s+2} + \frac{C}{s+3} = \frac{1}{s(s+2)(s+3)}$$

$$A = 1/6, B = -1/2, C = 1/3$$

$$Y(s) = \frac{1}{6s} - \frac{1}{2(s+2)} + \frac{1}{3(s+3)}$$

$$y(t) = \left(\frac{1}{6} + \frac{1}{3} e^{-3t} - \frac{1}{2} e^{-2t} \right) u(t)$$

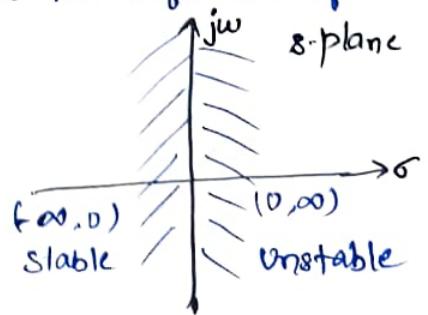
$$\lim_{t \rightarrow \infty} y(t) = \frac{1}{6} < \infty : \text{BIBO}$$

- In frequency domain, if all poles lie on the left side of jw -axis, then system is stable.

- At jw -axis, system is marginally stable.

- Marginally stable points are unstable system except for some points.

$$\begin{aligned} y(t) &= h(t) * u(t) \\ \alpha[y(t)] &= \alpha[h(t) * u(t)] \\ Y(s) &= H(s)U(s) \\ u(t) &= \delta(t) \Rightarrow U(s) = 1 \\ Y(s) &= H(s) \frac{1}{s} \\ \Rightarrow y(t) &= h(t) \end{aligned}$$



Eg. $\ddot{y}(t) + y(t) = u(t)$. find the step response & also consider $u(t)$.

Soln: Take LT both sides,

$$s^2 Y(s) + Y(s) = U(s)$$

$$\Rightarrow H(s) = \frac{1}{s^2 + 1} \rightarrow \text{poles: } \pm j \Rightarrow h(t) = \sin(t)$$

$$\therefore Y(s) = \frac{1}{s(s^2 + 1)} = \frac{1}{s} - \frac{s}{s^2 + 1}$$

$$\Rightarrow y(t) = u(t) - \cos(t) u(t) \\ = (1 - \cos t) u(t)$$

$\cos, \sin \in [-1, 1]$

↳ Bounded by 0 or 2 times $u(t)$.

$$\text{Let } u(t) = \sin t \Rightarrow U(s) = \frac{1}{1+s^2} \quad \text{bounded}$$

$$Y(s) = \frac{1}{(s^2 + 1)^2} \Rightarrow y(t) = t \cdot \sin t$$

$$\lim_{t \rightarrow \infty} y(t) = \infty \rightarrow \text{unbounded}$$

$$\text{Let } u(t) = 8 \sin 2t$$

$$U(s) = \frac{2}{s^2 + 4}$$

$$Y(s) = \frac{1}{(s^2 + 1)} \cdot \frac{2}{(s^2 + 4)} \Rightarrow y(t) = \underbrace{\cos 2t \pm 8 \sin 2t}_{\text{not exact}} \quad \text{↳ bounded}$$

Conclusion: This system goes to infinity if input natural frequency is same as the impulse response frequency at which pole occurs.

Eg $\ddot{y}(t) + \dot{y}(t) = u(t)$. ① $u(t) = \sin t$ and ② $u(t) = \text{unit-step}$.

$$s^2 Y(s) + s Y(s) = U(s)$$

$$\Rightarrow H(s) = \frac{1}{s+8^2} = \frac{1}{s(s+1)} ; \text{ Poles: } -1, 0$$

$$\Rightarrow h(t) = (1 - e^{-t}) u(t)$$

$$\textcircled{1} \quad Y(s) = H(s)U(s) = \frac{1}{s(s+1)} \cdot \frac{1}{s^2 + 1}$$

$$y(t) = 1 \pm e^{-t} \pm 8 \sin t$$

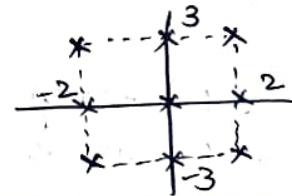
$$\lim_{t \rightarrow \infty} |y(t)| \leq M < \infty$$

$$\textcircled{2} \quad Y(s) = H(s)V(s) = \frac{1}{s(s+1)} \cdot \frac{1}{s} = \frac{1}{s^2(s+1)}$$

$$y(t) = t \pm e^{-t}$$

$$\lim_{t \rightarrow \infty} y(t) = \infty$$

- If pole is at origin, then DC i/p signal will make the system unstable.



Pole: $-2 \pm 3j$ (stable)

Pole: -2 (stable)

Pole: 2 (unstable)

Pole: $2 \pm 3j$ (unstable)

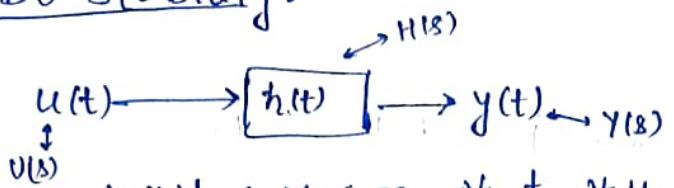


Pole: $3j$



Pole: 0



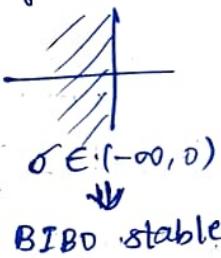
BIBO stability:

$$|u(t)| \leq M < \infty, \forall t, \forall u$$

$$|y(t)| \leq N < \infty, \forall t$$

$$H(s) = \frac{Y(s)}{U(s)} \quad \longleftrightarrow \quad \int_0^t |h(t)| dt < \infty$$

Poles of the
TF should
strictly lie
left ~~of~~
side
of s -plane



BIBO stable

↳ Absolutely integrable
(for stability)

Taylor series
to linearize
a system
about a point.

Q2 Prove that if the real parts of the poles are strictly negative, then the LTI system is BIBO stable.

Poles: steady state/Stability

Zeros: Transient Response.

Effects of Zeros of TF on Output:

e.g. ① $H_1(s) = \frac{1}{(s+1)(s+10)}$, ② $H_2(s) = \frac{s+2}{(s+1)(s+10)}$

Analyze unit-step response.

$$\text{SOLN: } ① \quad y_1(s) = H_1(s) U(s) = \frac{1}{s(s+1)(s+10)} = \frac{A}{s} + \frac{B}{(s+1)} + \frac{C}{(s+10)}$$

$$= \frac{1}{10} \cdot \frac{1}{s} - \frac{1}{9} \cdot \frac{1}{s+1} + \frac{1}{90} \cdot \frac{1}{s+10}$$

$$y_1(t) = \left(\frac{1}{10} - \frac{1}{9} e^{-t} + \frac{1}{90} e^{-10t} \right) u(t)$$

$$\lim_{t \rightarrow \infty} y_1(t) = \frac{1}{10} < \infty$$

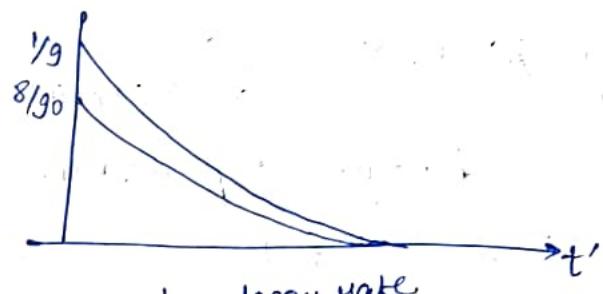
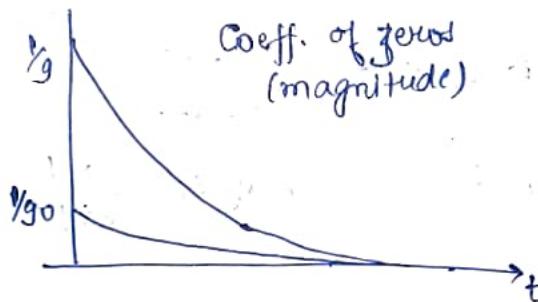
$$\textcircled{2} \quad Y_2(s) = H_2(s) U(s) = \frac{s+2}{s(s+1)(s+10)} = \frac{A}{s} + \frac{B}{s+1} + \frac{C}{s+10}$$

$$= \frac{2}{10} \cdot \frac{1}{s} + \frac{1}{9} \cdot \frac{1}{s+1} - \frac{8}{s+10} \cdot \frac{1}{90}$$

$$= \frac{1}{5} \frac{1}{s} + \frac{1}{9} \frac{1}{s+1} - \frac{8}{90} \cdot \frac{1}{s+10}$$

$$y_2(t) = \left(\frac{1}{5} - \frac{1}{9} e^{-t} - \frac{8}{90} e^{-10t} \right) u(t)$$

$$\lim_{t \rightarrow \infty} y_2(t) = \frac{1}{5} < \infty.$$



Dominant Poles: Poles closer to the jw axis (or origin).

↪ Less damping: Effects can be observed longer.

$\frac{(s+2)}{(s+1)(s+10)}$ \approx more towards the dominant pole ; Pole-zero cancellation.

$$H(s) = \frac{(s/2)}{(s-2)(s+1)(s+10)} \rightarrow \begin{array}{l} \text{3-degree} \\ \text{order: 3} \end{array} \rightarrow \begin{array}{l} \text{stable a/c to TF} \\ \text{unstable a/c to state-space} \end{array}$$

Minimal Realization of State-space

Controllability

Controllability: We should be able to move to any point from a point.

↪ Loss in degree of freedom \Rightarrow May not be able to get.

Observability: We need sensor to observe.

No. of poles : No. of energy storing elements.

Pole-zero cancellation \Rightarrow Reduction in degree of freedom.

Minimal Realization: when poles & zeros don't cancel.

In reality, ~~H*~~ system is unstable but as we cancel poles & zeros, it becomes stable.

Non-Minimum Phase Zeros (Nyquist plot)

Minimum Phase System: All zeros and poles must lie in the left half of s-plane.

Bode plot: Can only represent minm and non-minm phase sys.

Nyquist plot: Can represent both.

$$\text{Non-Minimum Phase system} = \text{All pass system} \times \text{Minm Phase system}$$

01-09-2025

Eg. ① $H_1(s) = \frac{s-2}{(s+1)(s+10)}$ ② $H_2(s) = \frac{-(s-1)}{(s+1)^2}$

Analyze step response of both. ③ $H_3(s) = \frac{s-1}{(s+1)^2}$

Soln: ① Poles: -1, -10; Zeros: 2
n=2; m=1.

$$H_1(s) = \frac{Y_1(s)}{U(s)} \Rightarrow Y_1(s) = H_1(s) U(s) = \frac{s-2}{s(s+1)(s+10)} = \frac{A}{s} + \frac{B}{s+1} + \frac{C}{s+10}$$

$$= \left(-\frac{2}{10}\right)\frac{1}{s} + \frac{3}{9} \cdot \frac{1}{s+1} - \frac{12}{90} \cdot \frac{1}{s+10}$$

$$y_1(t) = \left(-\frac{2}{10} + \frac{3}{9} e^{-t} - \frac{12}{90} e^{-10t}\right) u(t)$$

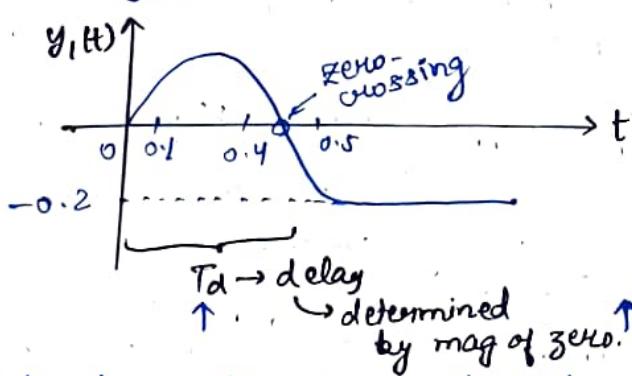
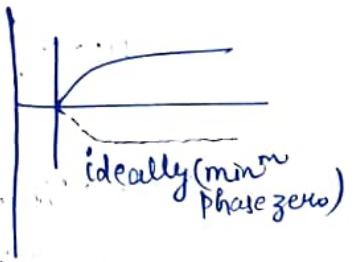
$$\lim_{t \rightarrow \infty} y_1(t) = -\frac{2}{10} < 0 \rightarrow \text{stable}$$

$$y_1(0) = 0, y_1(5) \approx -0.198$$

$$y_1(10) \approx -0.2$$

$$y_1(0.1) \approx 0.05, y_1(0.4) \approx 0.019$$

$$y_1(0.5) \approx -0.0007$$



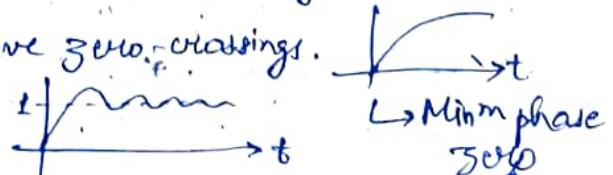
→ Non-minm phase zero
↳ called as faulty sys.

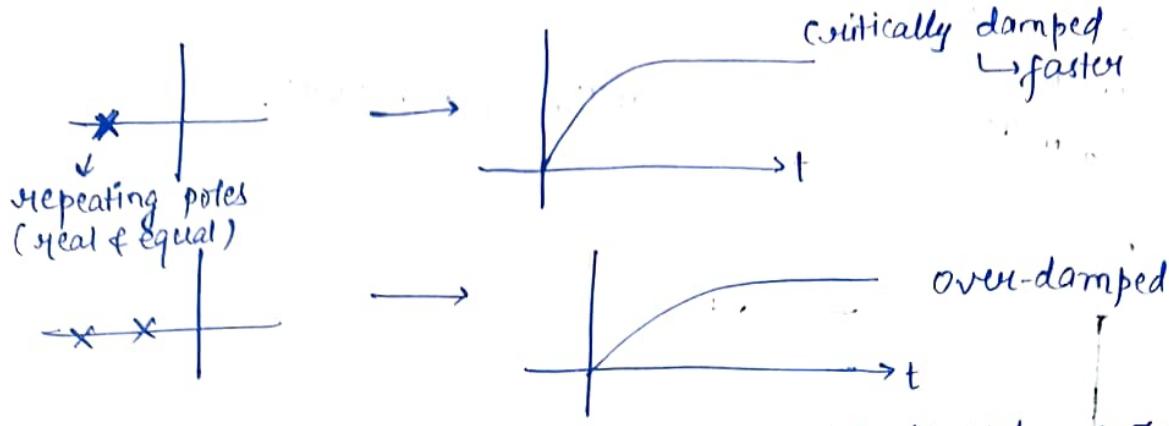
→ first go in the wrong dirn, then in reverse dirn.

↳ Should have been in only one-dirn.

→ Underdamped systems don't have zero-crossings.

↳ Oscillates around '1'.





- We design underdamped system with ξ value 0.7 - 0.8.
 → Time-delay in a system might introduce NMP zero.

Eg. $y(t) + y(t) = u(t - T_d)$, where $T_d > 0$ is transport delay

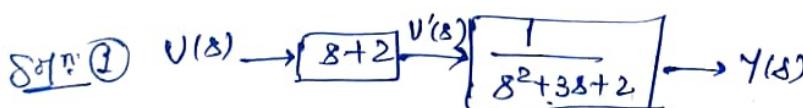
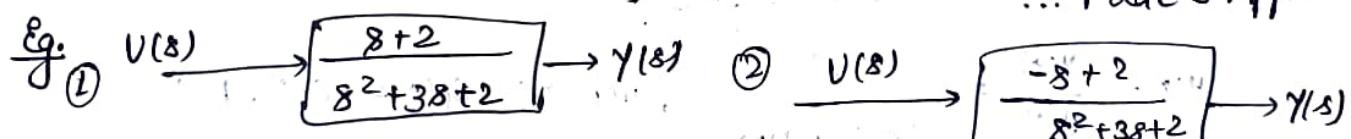
$$s y(s) + y(s) = e^{-s T_d} u(s) \Rightarrow H(s) = \frac{y(s)}{u(s)} = \left(\frac{e^{-s T_d}}{s+1} \right)$$

Exponential delay term can be approximated.

$$\textcircled{1} \quad e^{-s T_d} \approx \frac{1}{T_d s + 1} \Rightarrow H(s) = \frac{1}{(s+1)(T_d s + 1)}$$

$$\textcircled{2} \quad e^{-s T_d} = \frac{e^{-s T_d/2}}{e^{s T_d/2}} \approx \frac{1 - T_d s/2}{1 + T_d s/2} = \frac{2 - T_d s}{2 + T_d s}$$

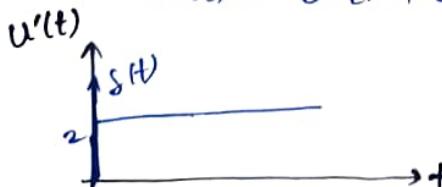
... Pade's Approx.



$$\frac{V'(s)}{V(s)} = s+2 \Rightarrow V'(s) = (s+2)V(s) \xrightarrow{\text{ILT}} u'(t) = \frac{d}{dt} (u(t)) + 2u(t)$$

Let's consider $u(t)$ to be step input.

$$\Rightarrow u'(t) = \delta(t) + 2u(t)$$



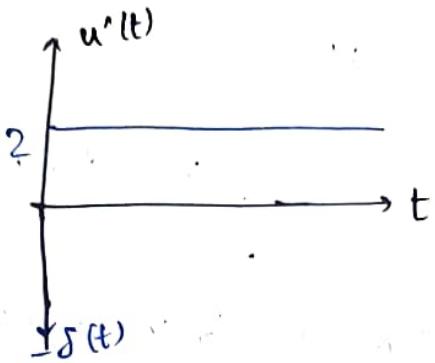
$\delta(t) \rightarrow$ shock
 ↳ in over dgn.

Poles: -1, -2
 Zeros: -2.

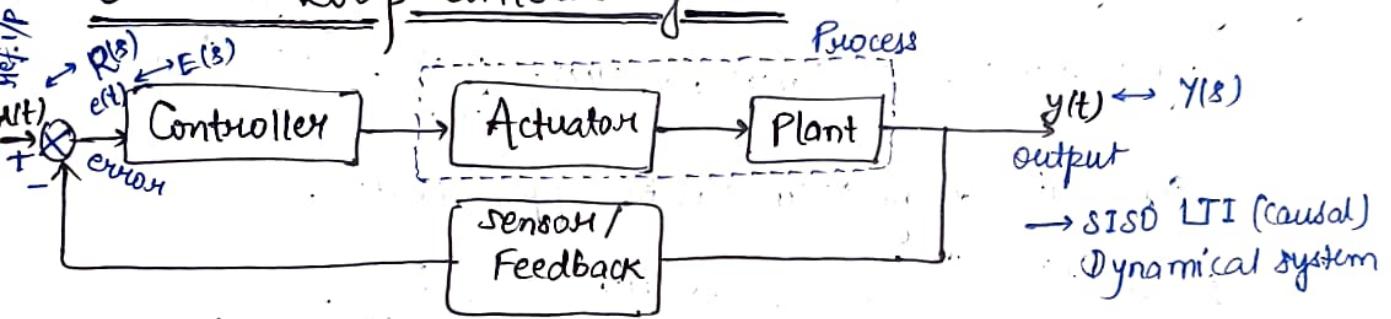
② Poles: -2, -1

Zeros: +2

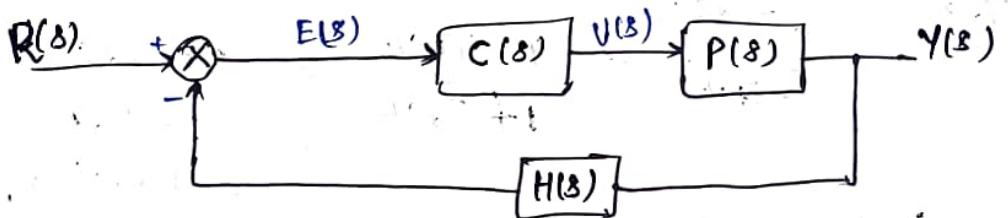
$$u(t) = -\delta(t) + 2u(t)$$



Closed Loop Control System



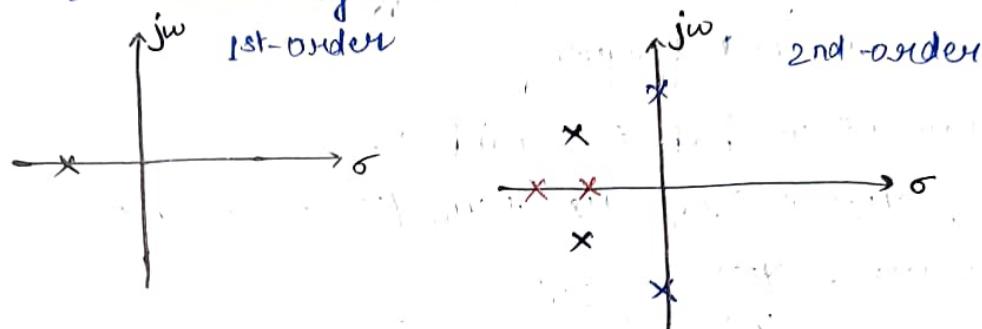
→ SISO LTI (causal)
Dynamical system



$$H(s) = \frac{Y(s)}{R(s)} = \frac{G(s)}{1 + G(s) H(s)}, \text{ where } G(s) = C(s) \cdot P(s)$$

- Full-state feedback.
- SISO LTI (causal) dynamic system.

- 1st-order system
- 2nd-order system
- Any higher order system can be studied through 1st and 2nd order systems.



- In first order, pole will always be at σ -axis only, and no complex poles, because for complex poles, its conjugate should also exist (2nd order).

1st-order system:

$$T \frac{dy(t)}{dt} + y(t) = u(t), \quad T > 0$$

Taking LT on both sides with ZIC,

$$\begin{aligned} T s Y(s) + Y(s) &= U(s) \\ \Rightarrow Y(s) &= \frac{U(s)}{1+Ts} \\ &= \frac{1}{s} \cdot \frac{1}{s+1/T} \\ &= \frac{1}{s} - \frac{1}{s+1/T} \end{aligned}$$

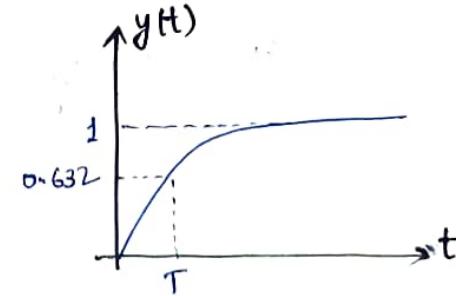
$$\stackrel{\text{ILT}}{y(t)} = (1 - e^{-t/T}) u(t)$$

$$\text{At } t=0, y(0)=0$$

$$t \rightarrow \infty, y(\infty)=1$$

$$t=T, y(T)=0.632$$

$$= 63.2\% \text{ of F.V.}$$



2nd-order System

$$\frac{d^2y(t)}{dt^2} + 2\xi\omega_n \frac{dy(t)}{dt} + \omega_n^2 y(t) = \omega_n^2 u(t),$$

where $y(t)$: o/p,
 $u(t)$: i/p

ξ : damping ratio / const. / coeff.

ω_n : natural frequency.

Take LT on both sides with LTC

$$s^2 Y(s) + 2\xi\omega_n s Y(s) + \omega_n^2 Y(s) = \omega_n^2 U(s)$$

$$\Rightarrow H(s) = \frac{Y(s)}{U(s)} = \frac{\omega_n^2}{s^2 + 2\xi\omega_n s + \omega_n^2}$$

Poles: $s^2 + 2\xi\omega_n s + \omega_n^2 = 0$

$$\Rightarrow s = -\xi\omega_n \pm \sqrt{\xi^2\omega_n^2 - \omega_n^2}$$

$$= -\xi\omega_n \pm \sqrt{\omega_n^2(\xi^2 - 1)}$$

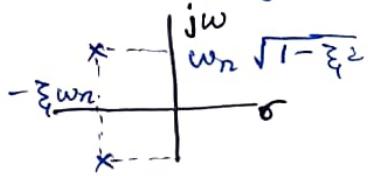
Case: (i) $\xi < 1$:

$$s = -\xi\omega_n \pm j\omega_n\sqrt{1-\xi^2} \rightarrow \text{complex poles}$$

oscillating with frequency,

$$\omega_d = \omega_n \sqrt{1-\xi^2} \rightarrow \text{damping freq.}$$

"Underdamped"



Case: (ii) $\xi = 1$:

$$s = -\omega_n, -\omega_n \rightarrow \text{Repeating real poles}$$

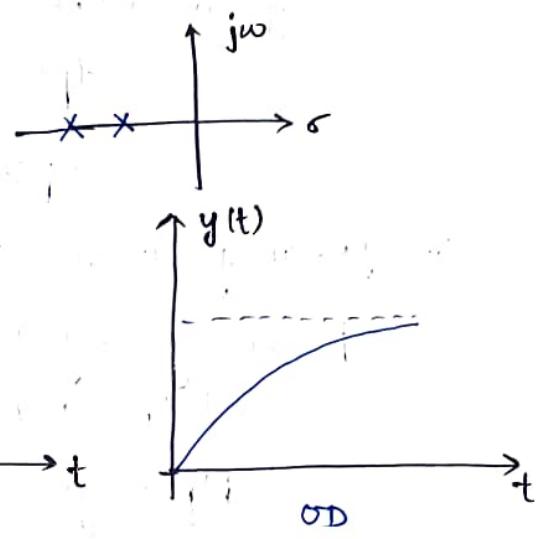
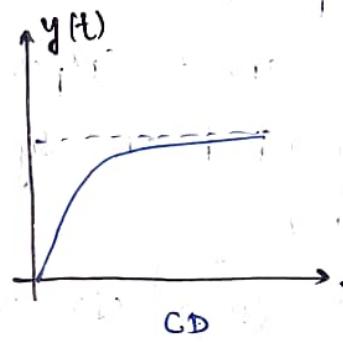
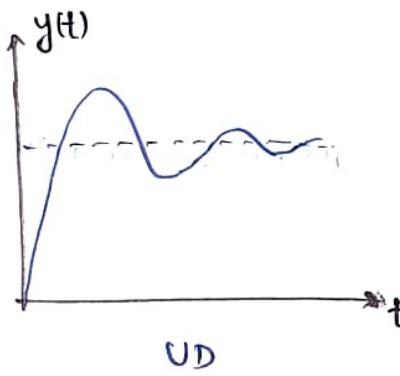
"Critically damped"



Case-III : $\xi > 1$:

$s = -\xi \omega_n \pm j\omega_d$: Non-repeating real poles

"overdamped"



Unit-step response of 2nd order system:

$$H(s) = \frac{\omega_n^2}{s^2 + 2\xi\omega_n s + \omega_n^2}$$

$$\Rightarrow Y(s) = \frac{\omega_n^2}{s(s^2 + 2\xi\omega_n s + \omega_n^2)}$$

$$= \frac{A}{s} + \frac{B s + C}{s^2 + 2\xi\omega_n s + \omega_n^2}$$

$$y(t) = 1 - e^{-\xi\omega_n t} \cos(\omega_d t) - \frac{\xi}{\sqrt{1-\xi^2}} e^{-\xi\omega_n t} \sin(\omega_d t)$$

$$t=0, y(0) = 1 - 1 - 0 = 0$$

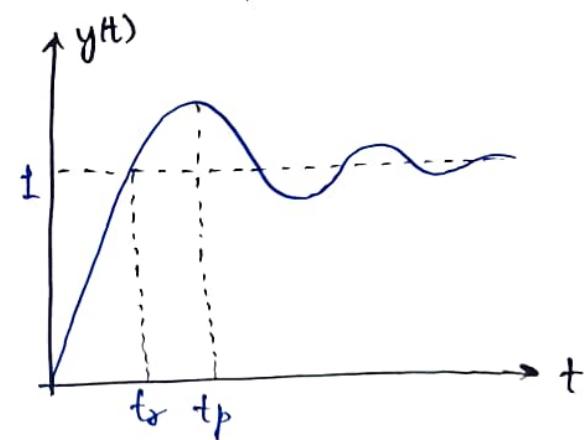
$$t \rightarrow \infty, y(\infty) = 1$$

i) Rise time, $t_r = \frac{\pi - \beta}{\omega_d}$

where,

$$\beta = \cos^{-1}(\xi)$$

$$\omega_d = \omega_n \sqrt{1-\xi^2}$$



↳ Time taken by the system (step) response to reach 100% of its final value for the 1st time.

(ii) Peak time (t_p):

: Time taken to reach the maximum value of the system response.

$$t_p = \frac{\pi}{\omega_d}$$

$$t_p > t_r$$

(iii) Max^m peak overshoot ($M_p\%$):

Percentage by which the system response max^m value exceed the F.V.

$$M_p = e^{-\left(\frac{\zeta_1 \pi}{\sqrt{1-\zeta_1^2}}\right)} \times 100\%$$

(iv) Setting time (t_s):

Time taken by the system response to reach FV within a particular band (2% and 5%).

$$t_s = \frac{4}{\zeta \omega_n} \quad (\pm 2\% \text{ band})$$

$$t_s = \frac{3}{\zeta \omega_n} \quad (\pm 5\% \text{ band})$$

Routh-Hurwitz Stability Criterion

$$H(s) = \frac{G_1(s)}{1 + G_1(s) H(s)}$$

Poles: $1 + G_1(s) H(s) = 0$

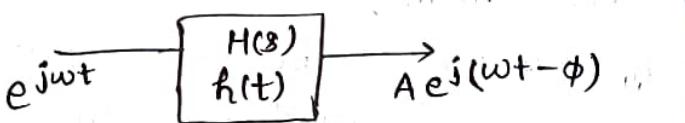
⇒ Poles: s^n roots of this eqn.

General eqn:

$$H(s) = \frac{N(s)}{D(s)}$$

$$D(s) = a_0 s^n + a_1 s^{n-1} + \dots + a_n \quad (n^{\text{th}})$$

Hurwitz Polynomial: Polynomial whose roots have negative real parts.



→ Just scaling and some phase is introduced.

Hurwitz Matrix: Matrix whose eigen value have negative real part.

$$D(s) = a_0 s^n + a_1 s^{n-1} + a_2 s^{n-2} + \dots + a_n$$

$$n=1 \Rightarrow D(s) = a_0 s + a_1 \Rightarrow s = -\frac{a_1}{a_0}$$

[a_1 & a_0 have same sign]

$$n=2 \Rightarrow D(s) = a_0 s^2 + a_1 s + a_2$$

$$\Rightarrow s = -\frac{a_1 \pm \sqrt{a_1^2 - 4 a_0 a_2}}{2 a_0}$$

$$= -\frac{a_1}{2 a_0} \pm \frac{\sqrt{a_1^2 - 4 a_0 a_2}}{2 a_0}$$

$$y = A \tilde{x} = \lambda x$$

↑ transformation (scalar)

$$A \in \mathbb{R}^{2 \times 3}$$

$$x \in \mathbb{R}^3$$

vector has not changed directly, only magnitude changed.

Eigen pair (λx):

Dim of linear transformation that never changes from original dim; but the scaling dim may change, defined by eigen value.

So, here $\frac{a_1}{2a_0} > 0$ (a_1 and a_0 of same sign)

$$\sqrt{a_1^2 - 4a_0a_2} < a_1 \Rightarrow a_0 \text{ & } a_2 \text{ of same sign.}$$

$\Rightarrow a_0, a_1$ and a_2 are of same sign.

$$n=3 \Rightarrow D(s) = a_0s^3 + a_1s^2 + a_2s + a_3$$

$$\text{Let } D_1(s) = s^3 + 2s^2 + 3s + 1 \quad \cancel{\text{Hence}}$$

$$\Rightarrow s = -0.43, -0.785 \pm 1.13j$$

\hookrightarrow This is Hurwitz polynomial.

$$D_2(s) = s^3 + 2s^2 + s + 1 \quad \cancel{\text{Hence}}$$

$$\Rightarrow s = -2.17, \quad \cancel{0.09} \pm 1.17j$$

\hookrightarrow Not Hurwitz, +ve real part.

$$D_3(s) = s^3 + 2s^2 + 3s \quad \cancel{\text{Hence}}$$

$$\Rightarrow s = 0, -1 \pm \sqrt{2}j$$

\hookrightarrow Not Hurwitz

For $n \geq 3$, All the coefficients have to be non-negative.

\hookrightarrow This is necessary but not sufficient condition.

Necessary condition: All coefficients must be non-zero and of the same sign!!

Sufficient: Routh Array

s^n	a_0	a_2	$a_4 \dots$
s^{n-1}	a_1	a_3	$a_5 \dots$
s^{n-2}	$b_1 = \frac{a_1a_2 - a_0a_3}{a_1}$	b_2	$b_3 \dots$
\vdots			
s^0			

$$D(s) = a_0s^n + a_1s^{n-1} + a_2s^{n-2} + a_3s^{n-3} + \dots + a_n$$

$$b_2 = \frac{a_1a_4 - a_0a_5}{a_1}$$

* Sufficient: Along the 1st column, all coefficients cond'n are positive.

* If there are sign changes, the polynomial is not Hurwitz & corresponding system is unstable.

* No. of sign changes denote the no. of pole in +ve side of jw-axis.

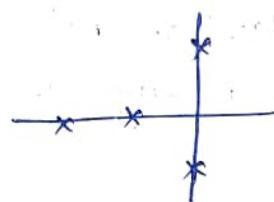
Case-II : The 1st element in any row is zero with other elements as non-zero.

Eq. ① $D(s) = s^4 + 3s^3 + 3s^2 + 3s + 2 \Rightarrow$ Roots: $-1, -2, \pm j$

s^4	1	3	3	2
s^3	3	3		
s^2	$\frac{3 \times 3 - 3 \times 1}{3} = 2$	12		
s^1	$\theta \rightarrow \epsilon$	No sign change		
s^0	2			$\epsilon \rightarrow 0, \epsilon > 0$

→ Marginally stable (due to zero).

~~Destabilized~~



Case-II:

$$D(s) = s^7 + 6s^6 + 11s^5 + 6s^4 + 4s^3 + 24s^2 + 44s + 24$$

s^7	1	11	4	44
s^6	6	6	24	24
s^5	10	0	40	
s^4	6	0	24	
s^3	0	4	0	
s^2	$\theta \rightarrow \epsilon$	24		

$$A(s) = 6s^4 + 24 = 0$$

$$\Rightarrow A(s) = s^4 + 4 = 0$$

$$A'(s) = \frac{dA}{ds} = 4s^3$$

Replace s^3 by 4.

s^1	-96/ ϵ
s^0	24

→ Entire row becomes zero.
↓
Construct auxiliary eqn.
(not ϵ)
with the row above.

$$s^2 \rightarrow s^1 \text{ (1 sign change)}$$

$$s^1 \rightarrow s^0 \text{ (Another sign change)}$$

⇒ Unstable system

$$n = \text{No. of roots/poles} = 7$$

2 sign changes \Rightarrow 2 poles on RHP.

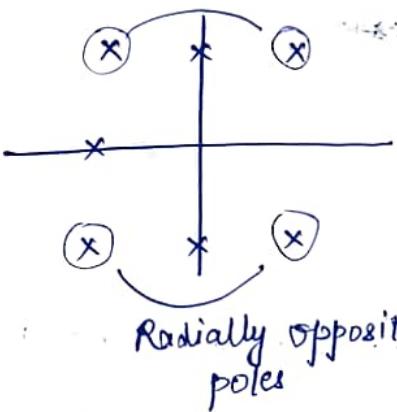
$$A(s) = s^2 + 4 = 0 \Rightarrow A'(s) = \frac{dA}{ds} = 4s^3$$

$$\Rightarrow (s^2 + 2s + 2)(s^2 - 2s + 2) = 0$$

$i \pm j$

$1 \pm j$

⇒ 4 radially opposite poles ;
1 pole on LHP.



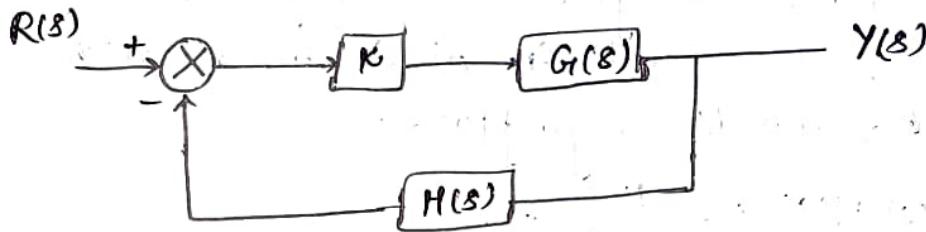
All zeros in a row, ensure
atleast one pair of poles
on the Imag-axis.

ROOT LOCUS..

Closed Loop TF:

$$H(s) = \frac{G(s)}{1 + G(s) H(s)}$$

Characteristic eqn: $1 + G(s) H(s) = 0 \Rightarrow$ soln/roots are CL poles



$$\text{CLTF} = \frac{K G(s)}{1 + K G(s) H(s)}$$

$\xrightarrow[K=0]{\delta_1}$ $\xrightarrow[K \rightarrow \infty]{\delta_1}$

$\exists K > 0$, for which δ_1 is CL pole.

$1 + G(s) H(s) = 0$: charac. eqn

$\hookrightarrow G(s) H(s)$: OL gain/TF

$\hookrightarrow G(s) H(s) = -1$: OLTF

$|G(s) H(s)| = 1$... Magnitude Criteria

$\angle G(s) H(s) = \pm 180^\circ (2k+1)$, $k=0, 1, \dots$

(odd multiples of $\pm 180^\circ$)

... Angle criteria

OLTF: $G(s) H(s) = \frac{K(s+z_1)(s+z_2) \dots (s+z_m)}{(s+p_1)(s+p_2) \dots (s+p_n)}$

m: no. of zeros

n: no. of poles

How would the CL poles change as we vary $K \geq 0$?

\hookrightarrow Answer lies in RL representation.

$$CL \text{ char. eqn: } 1 + G(s) H(s) = 1 + \frac{K(s+z_1)(s+z_2)\dots(s+z_m)}{(s+p_1)(s+p_2)\dots(s+p_n)} = 0$$

Branch: Trajectories of RL

- n = no. of branches. ($=$ no. of CL poles)
- Branches always start from pole locations, and terminate at zeros or move towards asymptotes.
- n branches always start from poles, terminate at ' m ' zeros and ~~($n-m$)~~ ($n-m$) branches move towards asymptotes.

Eg. OLT: $G(s) H(s) = \frac{K}{(s+3)(s-2)}$

$$n=2; p_1=-3, p_2=2$$

$$m=0$$

- 2 branches; 2 branches towards asymptotes.

$$CL \text{ char. eqn: } 1 + G(s) H(s) = 0$$

$$\Rightarrow 1 + \frac{K}{(s+3)(s-2)} = 0$$

$$\Rightarrow (s+3)(s-2) + K = 0$$

$$K=0 \Rightarrow s=-3, 2$$

Eg. $G(s) H(s) = \frac{K(s+1)}{(s+3)(s-2)}$

$$CL \text{ CE: } 1 + G(s) H(s) = 0$$

$$\Rightarrow 1 + \frac{K(s+1)}{(s+3)(s-2)} = 0$$

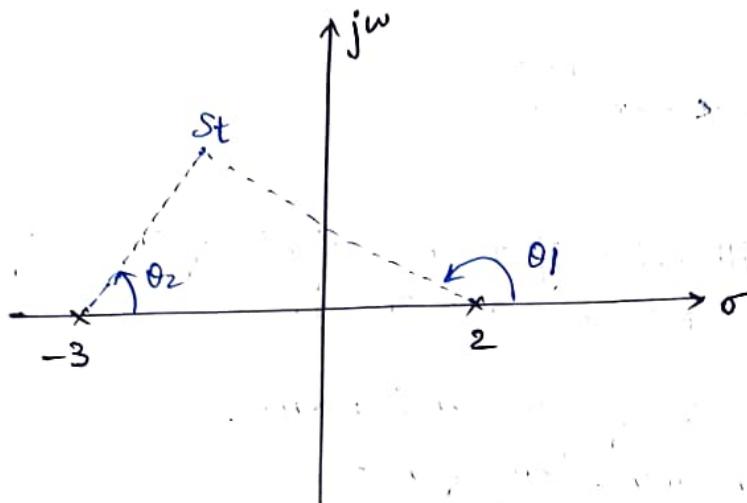
$$\Rightarrow (s+3)(s-2) + K(s+1) = 0$$

$$\Rightarrow \frac{1}{K}(s+3)(s-2) + (s+1) = 0$$

$$\text{if } K \rightarrow \infty, s = -1.$$

How to check if a test point ' s_t ' lies in R.L?
 ↳ ' s_t ' is a CL pole for a certain $K > 0$ value.

Eg $G(s) H(s) = \frac{K}{(s+3)(s-2)}$



$$|G(s_t) H(s_t)| = \left| \frac{K}{(s_t + 3)(s_t - 2)} \right| = 1 \text{ (mag. criteria)}$$

whether ' s_t ' satisfies this.

$$\angle G(s_t) H(s_t) = \angle K - \angle(s_t + 3) - \angle(s_t - 2)$$

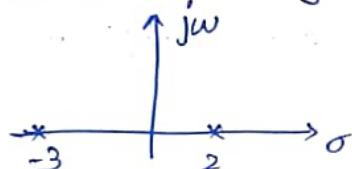
$$= 0^\circ - \theta_1 - \theta_2$$

$$= -(\theta_1 + \theta_2) = \pm 180^\circ (2K + 1)$$

whether ' s_t ' satisfies this.

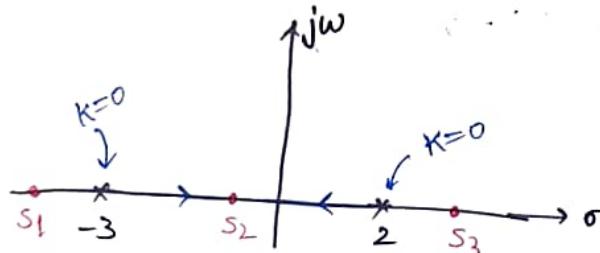
Steps:

① Locate the OL poles and zeros.



$n=2 \Rightarrow 2$ branches

$m=0 \Rightarrow (n-m)=2$ branches towards asymptotes.



For $s_1: (-\infty, -3) : 360^\circ \times$

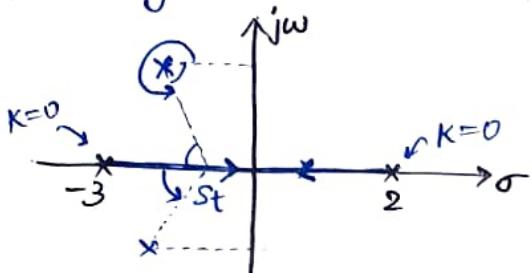
for $s_2: [-3, 2] : 180^\circ \checkmark \rightarrow$ Part of RL

for $s_3: (2, \infty) : 0^\circ \times$

② Locate the RL on the real axis.

↳ Determined by real OL poles and zeros.
(others get cancelled to $360^\circ/0^\circ$)

12-09-2025



- RL branch will exist if to the right of the test point, odd no. of poles + zeros are there.

③ Find the asymptotes of root loci ($n > m$).

n : no. of branches/no. of poles

m : termination of branches

$n-m$: no. of asymptotes

$$\angle \text{asymptotes} = \pm \frac{180^\circ (2k+1)}{n-m}, k=0, 1, \dots$$

Centroid: intersection of asymptotes with real axis.

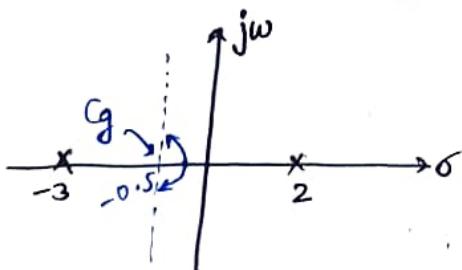
$$C_g = \frac{\sum \text{Real parts of poles} - \sum \text{Real parts of zeros}}{n-m}$$

$$\text{Ex. } G(s) H(s) = \frac{k}{(s+3)(s-2)}$$

$$\theta_{\text{asym.}} = \pm \frac{180^\circ (2k+1)}{2} = \pm 90^\circ (2k+1), k=0, 1, \dots$$

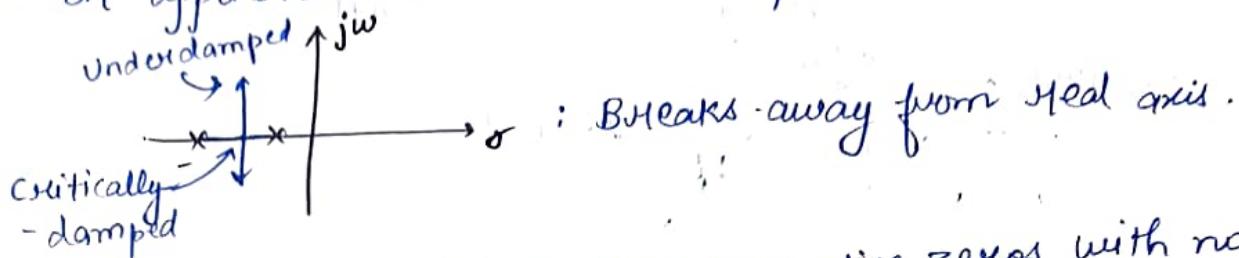
$$= \pm 90^\circ$$

$$C_g = \frac{(-3+2)-0}{2} = -0.5$$

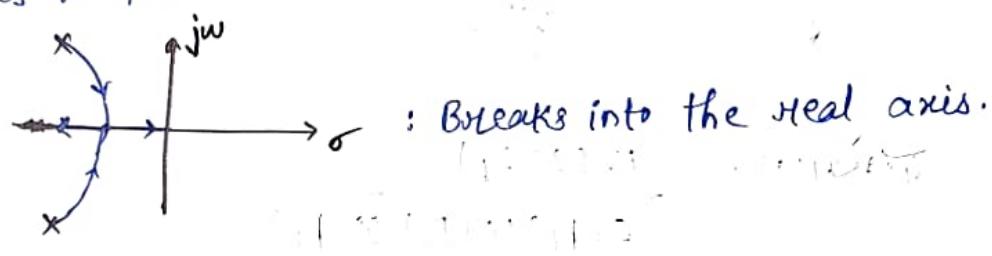


④ Determine breakaway / break-in points.

- BA applicable to two consecutive poles with no zero in b/w.



- BI points applicable to two consecutive zeros with no poles in b/w.



e.g. $P_1(s) = (s+2)(s+3) \Rightarrow P_1'(s) = (s+2) + (s+3)$

$$P_2(s) = (s+2)^2(s+3) \Rightarrow P_2'(s) = 2(s+2)(s+3) + (s+2)^2$$

Sol'n: $-2, -2, -3$

Sol'n: $-2, -3$

$$P_1'(s)|_{s=-2, -3} \neq 0$$

$$P_2'(s)|_{s=-2} = 0$$

- At BA / BI points, double solutions exist and hence the derivative becomes zero.

e.g. $G(s)H(s) = \frac{K}{(s+3)(s-2)} = \frac{KA(s)}{B(s)}$

CL char. eqn: $1 + G(s)H(s) = 0$

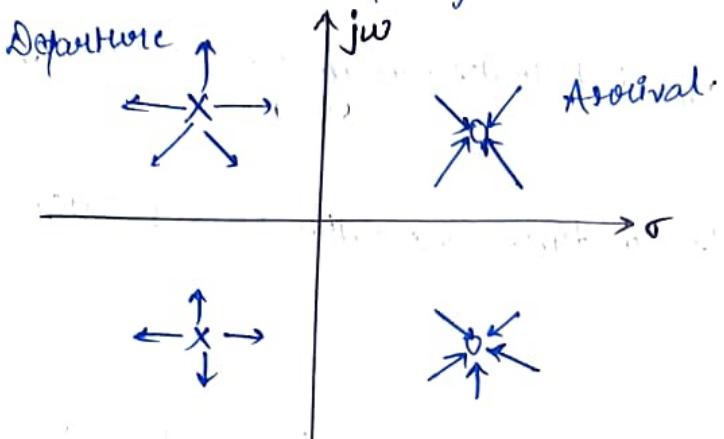
$$\Rightarrow 1 + \frac{KA(s)}{B(s)} = 0 \triangleq P_{CL}(s)$$

$$\frac{dP_{CL}(s)}{ds} = \frac{B(s)A'(s) - A(s)B'(s)}{(B(s))^2} = 0 \quad (K > 0)$$

$$P_{CL}'(s) = B(s)A'(s) - A(s)B'(s) = 0$$

\hookrightarrow soln will give s_{BA} / s_{BI} point.

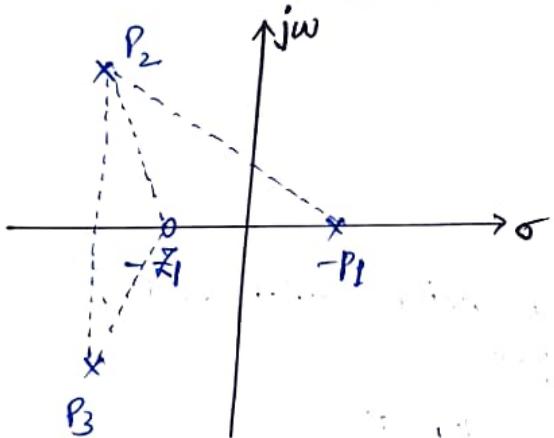
⑤ Find the angle of departure / angle of arrival.



$$\text{Eq. } G(s) H(s) = \frac{K(s + Z_1)}{(s + P_1)(s + P_2)(s + P_3)}$$

where P_2 and P_3 are complex poles.

$$Z_1 > 0, P_1 < 0.$$



$\angle \text{Departure} = 180^\circ - (\text{sum of angles made by vectors from other OL poles}) + (\text{sum of angles made by vectors from other OL zeros to this pole})$

$\angle \text{Arrival} = 180^\circ - (\text{sum of angles made by vectors from other OL poles}) + (\text{sum of angles made by vectors from other OL zeros to this zero}).$

$\angle \text{Departure from } P_2:$

$$\angle G(s) H(s) = \pm 180^\circ (2K+1), K=0, 1, \dots$$

$$\underbrace{\pm 180^\circ(2K+1)}_{\text{Take } 180^\circ} = \underbrace{\angle K + \angle 8 + \angle I}_{0^\circ} - (\underbrace{\angle S + P_1 + \angle 8 + P_2}_{\theta_{\text{dep}}} + \underbrace{\angle S + P_3}_{P_2})$$

$$\Rightarrow \theta_{\text{dep}}^{-P_2} = 180^\circ + \angle -P_2 + z_I - (\angle -P_2 + P_1 + \angle -P_2 + P_3) \quad [\text{Replace } s \text{ with } -P_2]$$

Since RL is always symmetric about axes:

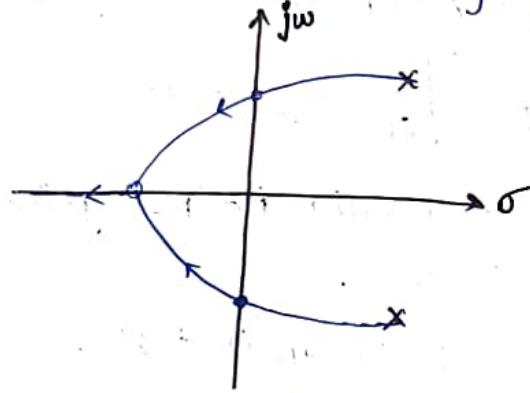
Thus,

$$\theta_{\text{dep}}^{1-P_2} = -\theta_{\text{dep}}^{-P_3}$$

Eg. $G(s) H(s) = \frac{K}{(s+3)(s-2)} \rightarrow \text{No } \theta_{\text{dep}}, \text{ no } \theta_{\text{var}}$.

⑥ Determine the cross-over points.

↳ Tells about the stability.



Put $s=j\omega$ in char. eqn. and find ω and K values.

Char. eqn: $1 + G(s) H(s) = 0$

$$\Rightarrow 1 + \frac{K}{(s+3)(s-2)} = 0$$

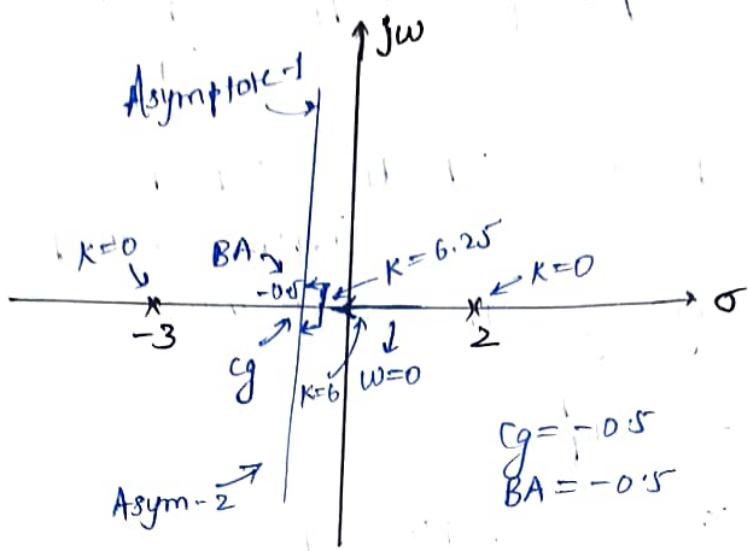
$$\Rightarrow s^2 + 8 - 6 + K = 0$$

$$s=j\omega \Rightarrow -\omega^2 + j\omega + (K-6) = 0$$

Comparing Real and Imag parts

$$\Rightarrow K-6-\omega^2=0, \omega=0$$

$$\Rightarrow K=6$$



$$1 + G(s) H(s) = \frac{KA(s)}{B(s)} + 1 = 0 \Rightarrow K = -\frac{B(s)}{A(s)}$$

BA point: $A'(s) B(s) - A(s) B'(s) = 0$ $\left[\frac{dK}{ds} = 0 \right]$

$$K = -\frac{(s+3)(s-2)}{1} \Big|_{s_b=-0.5} = 6.25 > 0$$

If $K \in \mathbb{R}^+$, then s_b is a valid BA/BI point,
provided s_b itself is real.

For stability, $K > 6$: CL system is stable.

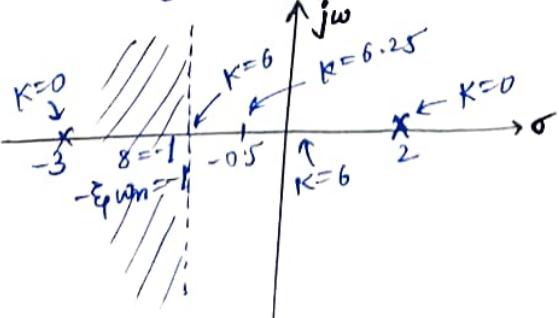
Q) Find the range of various values (parameters) for which the previous system has maxm settling time (In 2% band) of $4s$, considering the system to be underdamped.

Sol: $\zeta_s \leq 4s \Rightarrow \frac{4}{\zeta w_n} \leq 4 \Rightarrow \zeta w_n \geq 1 \Rightarrow -\zeta w_n \leq -1$.

Underdamped $\Rightarrow \zeta < 1$

$$\Rightarrow s_{1,2} = -\zeta w_n \pm j w_n \sqrt{1-\zeta^2}$$

$$\Rightarrow \text{Re}(s_1, s_2) = -\zeta w_n \leq -1$$



$$\text{Char. eqn: } 1 + \frac{1}{(8+3)(s-2)} = 0$$

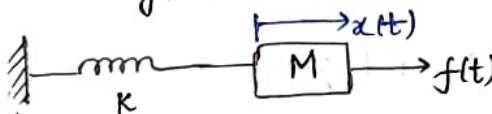
$$\Rightarrow s^2 + s - 6 + K = 0$$

$$\text{Put } s=1 \Rightarrow 1 - 1 - 6 + K = 0 \\ \Rightarrow K = 6$$

17-09-2025

Eg. Consider the arrangement shown. Design a suitable controller in ve feedback to stabilize the CL system.

Consider $M = 1 \text{ kg}$, $K = 1 \text{ N/m}$. Sketch RL.



Governing eqn of the mass-spring system:

$$M \frac{d^2x(t)}{dt^2} + Kx(t) = f(t)$$

Take LT on both sides with XIC,

$$M s^2 X(s) + KX(s) = F(s)$$

$$\Rightarrow \frac{X(s)}{F(s)} = \frac{1}{s^2 + K} = \frac{1}{s^2 + 1} \quad (\text{OLTF})$$

We have to design a PD controller.

PID:

P: Present

I: Past

D: Future

$$\alpha \left(\frac{dx}{dt} \right) \xrightarrow{\text{LT}} sX(s)$$



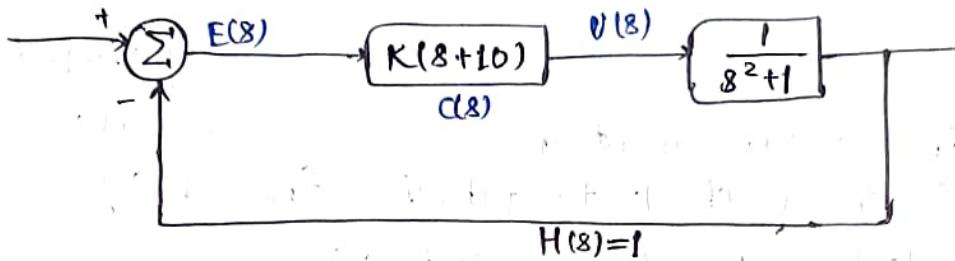
$$\text{PD: } \frac{U(s)}{E(s)} = \frac{K_p}{P} + \frac{s K_D}{D}$$

$$C(s) = (K_p + K_D s)$$

Let $K_p = 20$, $K_D = 2$.

$$C(s) = K_p \left(1 + \frac{8K_D}{K_p} \right)$$

$$= K (s + 10)$$

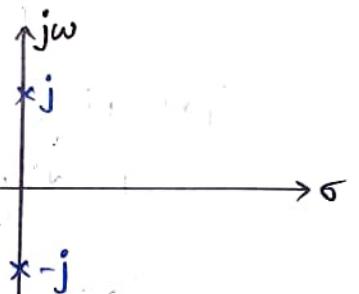


RL:

Step-1: Locate OL poles and zeros.

$$G(s) = \frac{K(s+10)}{s^2 + 1}$$

$$G(s)H(s) = \frac{K(s+10)}{s^2 + 1}$$



Step-2: Locate the RL on the real axis.

- Count the no. of poles and zeros right of the chosen point.

$(-\infty, -10]$: Part of RL ✓

$[-10, \infty)$: Not part of RL ✗

Step-3: Determine the asymptotes of RL.

$$\text{Here, } n_p = 2, m_z = 1$$

$$\text{No. of asymptotes} = n_p - m_z = 2 - 1 = 1$$

$$\angle \text{ asymptotes} = \pm \frac{180^\circ (2k+1)}{n_p - m_z}$$

$$= \pm 180^\circ \quad (k=0)$$

only 2 poles

$$\text{centeroid, } C_g = \frac{\sum R_E(\text{poles}) - \sum R_E(\text{zeros})}{n_p - m_z}$$

$$= \frac{0 - (-10)}{1} = 10.$$

Step-4: Determine BA/BI points.

$$-A(s)B'(s) + A'(s)B(s) \Big|_{s=s_b} = 0$$

where $A(s) = (s+10)$
 $B(s) = (s^2+1)$

$$\Rightarrow -(s+10) \cdot 2s + 1 \cdot (s^2+1) = 0$$

$$\Rightarrow -s^2 + 1 - 20s = 0$$

$$\Rightarrow s^2 + 20s - 1 = 0$$

$$\Rightarrow s = -20.05, \quad \checkmark \quad \times \rightarrow \text{not part of RL}$$

char. eqn: $1 + \frac{K(A(s_b))}{B(s_b)} = 0$

$$\Rightarrow K = -\frac{B(s_b)}{A(s_b)} = 40.1 \text{ rad/sec}$$

Step-5: Angle of departure

$$\angle \theta_{\text{dep}}^{+} = 180^\circ - \sum \angle \text{ made by other OL poles to this pole} \\ + \sum \angle \text{ made by other OL zeros to this pole.}$$

$$= 180^\circ - 90^\circ + \tan^{-1}(1/10) = 95.71^\circ$$

$$\angle \theta_{\text{dep}}^{-} = -95.71^\circ$$

Step-6: Find the crossover point.

CL char. eqn: $1 + \frac{K(s+i0)}{s^2+1} = 0$

$$\Rightarrow s^2 + Ks + 1 + i0K = 0$$

Put, $s = j\omega$

$$\Rightarrow -\omega^2 + jK\omega + 1 + i0K = 0$$

$$\Rightarrow (1 + i0K - \omega^2) + jK\omega = 0$$

Compare Real & Imag. part:

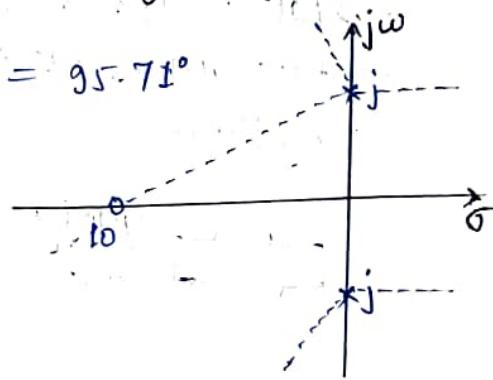
$$\begin{cases} K\omega = 0 \\ 1 + i0K - \omega^2 = 0 \end{cases}$$

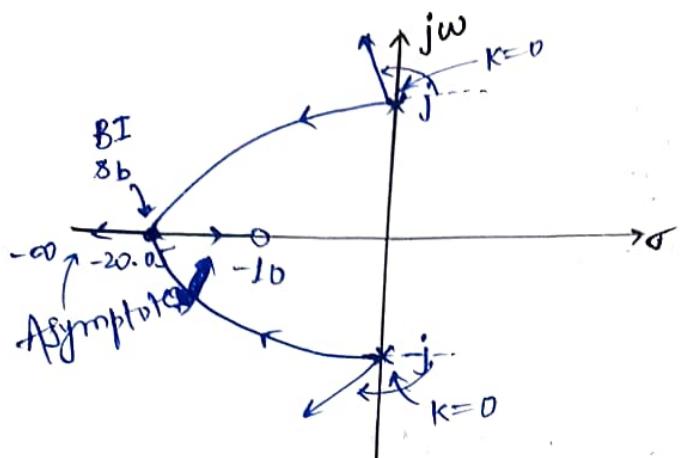
$\Rightarrow K=0$
 $\text{or } \omega=0$

If $K=0$, $\omega^2=1 \Rightarrow \omega = \pm 1$ [$K=0 \rightarrow \text{not valid}$]

If $\omega=0$, $K = -1/10$, but $K < 0$ not possible.

So, there are no crossover point except the pure imaginary poles.





$$\left. \frac{A'(s)B(s) - A(s)B'(s)}{s = s_{BA}/s_{BI}} \right| = 0$$

STATE SPACE

Space: Collection of points in certain dimension.

↪ Collection (set) of similar entities (points).

- State is related to dynamic systems.

- State represents the evolution characteristics of a dynamical system.

→ Dynamical system is represented
by mathematical equation.
↪ Set states.

State variables: Variables which can completely describe the evolution of a dynamical system.

↪ Minm no. of variables that completely characterize a dynamical system.

State Space: Collection/set of minm no. of variables that completely characterize a dynamical system.

$$\frac{dy}{dt} = f(y, t) \quad y = \dot{x} = \frac{dx}{dt} = f(x)$$

$f: \mathbb{R}^n \rightarrow \mathbb{R}^m$ → May or may not be linear, TI, SISO.

↪ fn/mapping.

($n \neq m$, $n, m > 0$)

- $y = Ax$ (Linear), $x, y \in \mathbb{R}^n$
- We'll deal with only linear control system.
 - Use Taylor Series expansion to linearize a system.

State-Space Representation:

$$\dot{x} = Ax + Bu$$

$$y = cx$$

$x \in \mathbb{R}^n$: state vector

$y \in \mathbb{R}^m$: output vector, $u \in \mathbb{R}^p$: input vector

$A \in \mathbb{R}^{n \times n}$: state matrix (or system matrix)

$B \in \mathbb{R}^{n \times p}$: Input matrix

$C \in \mathbb{R}^{m \times n}$: Output matrix,

A, B : system matrix

State

- Sensor: Observability

- Actuator: Controllability: Move from one state to another in finite time.

$$u = Kx, \quad K \in \mathbb{R}^{p \times n}$$

controller controller
 gain

$$\dot{x} = \underbrace{(A + BK)x}_{\text{linear}}$$

e.g. $H(s) = \frac{(s+2)}{(s+2)(s+3)(s+4)}$ → Pole-zero cancellation

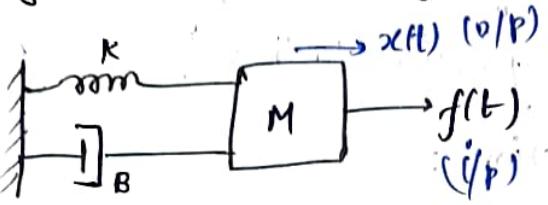
 $\lambda(A) = \{2, -3, -4\}$ → Doesn't lie!

22-09-2025

State-Space Representation

- Non-linear system
- Time varying
- MIMO
- Non-zero initial condition
- No pole-zero cancellation
- Internal state-based approach.

Mechanical system: Mass-Spring Damped (MSD) system



$$\text{MSD eqn of motion: } M\ddot{x}(t) + B\dot{x}(t) + Kx(t) = f(t) \dots \textcircled{1}$$

K: spring constant (Nm^{-1})

B: damping coefficient (Nm^{-1}s)

M: Mass (kg)

f(t) : Force (N) : Input

x(t) : Displacement (m) : Output

Take LT on both side with ZIC,

$$M\dot{s}^2 X(s) + B\dot{s} X(s) + KX(s) = F(s)$$

$$\Rightarrow \text{T.F.} = \frac{X(s)}{F(s)} = \frac{1}{M\dot{s}^2 + B\dot{s} + K}$$

Let $x_1(t) = x(t)$, [choose the variables]

$$x_2(t) = \dot{x}(t) = \dot{x}_1(t)$$

$$\textcircled{1} \Rightarrow M\ddot{x}_1(t) + Bx_2(t) + Kx_1(t) = f(t)$$

$$\Rightarrow \dot{x}_2(t) = -\frac{B}{M}x_2(t) - \frac{K}{M}x_1(t) + \frac{1}{M}f(t)$$

$$\dot{x}_1(t) = x_2(t)$$

$\dot{x} = Ax + Bu$
$y = cx$

$$x = \begin{bmatrix} x_1(t) \\ x_2(t) \end{bmatrix}$$

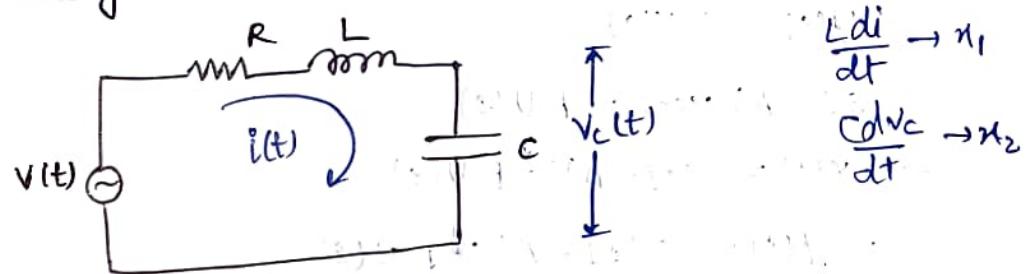
$$\dot{x} = \begin{bmatrix} \dot{x}_1(t) \\ \dot{x}_2(t) \end{bmatrix} = \underbrace{\begin{bmatrix} 0 & 1 \\ -\frac{K}{M} & -\frac{B}{M} \end{bmatrix}}_{\text{State matrix}} \begin{bmatrix} x_1(t) \\ x_2(t) \end{bmatrix} + \underbrace{\begin{bmatrix} 0 \\ \frac{1}{M} \end{bmatrix}}_{\text{I/P vector}} f(t)$$

$$y = x_1(t) = \begin{bmatrix} 1 & 0 \end{bmatrix} x$$

$$\# x(k+1) = Ax(k) + Bu(k)$$

$$y(k) = cx(k)$$

Electrical system: Series RLC circuit



$$\frac{d^2v_c}{dt^2} + \frac{1}{RC}v_c + \frac{1}{L}i = 0$$

$$\frac{d^2v_c}{dt^2} + \frac{1}{RC}v_c + \frac{1}{L} \cdot \frac{1}{R}v_c = 0$$

$$\frac{d^2v_c}{dt^2} + \left(\frac{1}{RC} + \frac{1}{RL} \right) v_c = 0$$

Homogeneous differential equation

Initial conditions: $v_c(0) = 0$, $\frac{dv_c}{dt}|_{t=0} = 0$

Solution: $v_c(t) = A \cos(\omega t) + B \sin(\omega t)$

Boundary conditions: $v_c(0) = 0 \Rightarrow A = 0$

Initial condition: $\frac{dv_c}{dt}|_{t=0} = 0 \Rightarrow B = 0$

Conclusion: $v_c(t) = 0$ for all t

Conclusion: $i(t) = 0$ for all t

Conclusion: $\frac{dv_c}{dt} = 0$ for all t

Conclusion: $\frac{d^2v_c}{dt^2} = 0$ for all t

SS to TF:

$$\begin{aligned}\dot{x} &= Ax + Bu & \dots & \textcircled{1} \\ y &= Cx & \dots & \textcircled{2}\end{aligned}$$

Taking LT on both with ZIC for $\textcircled{1}$ & $\textcircled{2}$,

$$\mathcal{X}(s) = Ax(s) + Bu(s)$$

$$y(s) = Cx(s)$$

$$\Rightarrow (sI - A)x(s) = Bu(s)$$

$$\Rightarrow x(s) = (sI - A)^{-1}Bu(s)$$

$$\therefore Y(s) = C(sI - A)^{-1}Bu(s)$$

$$TF = \frac{Y(s)}{U(s)} = C(sI - A)^{-1}B$$

$$P(s) = (sI - A) : \text{State transition matrix.}$$

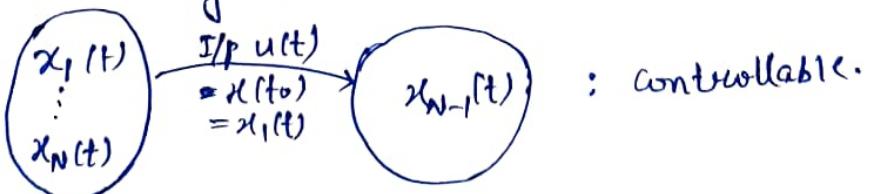
$$\# Ax = \lambda x$$

$$\Rightarrow (A - \lambda I)x = 0 \Rightarrow (\underbrace{\lambda I - A}_{})x = 0$$

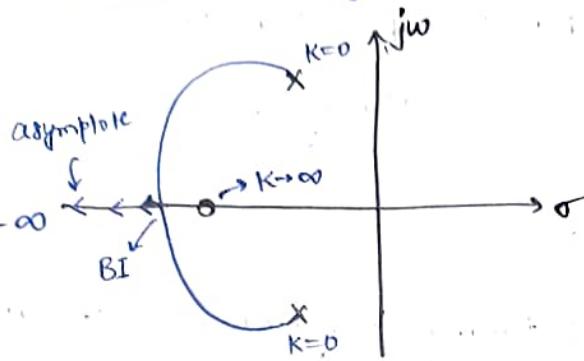
Controllability and Observability:

Controllability: If there exists an input to a system that can drive every state variable to a desired state value in finite time, then the system is called controllable.

Observability: If any state vector $x(t_0)$ can be found from measured o/p $y(t)$ over a finite interval time from t_0 , then the system is called observable.



Controllability and Observability:



Observability: Certain input value which can steer the state value.

OR
state values through actuation (A, B).

Observability: O/p, state (C, D)

(A, B, C) : System tuple/matrices

Controllability matrix:

$$\bar{C} = [B \ AB \ A^2B \ \dots \ A^{n-1}B]$$

n: Rank of A

Observability matrix:

$$\bar{O} = [C^T \ AT^T \ A^2T^T \ \dots \ (A^{n-1})^T C^T]$$

$$= \begin{bmatrix} C \\ CA \\ CA^2 \\ \vdots \\ CA^{n-1} \end{bmatrix}^T$$

$$\# \quad C = \begin{bmatrix} 2 & 1 \end{bmatrix}, \quad A = \begin{bmatrix} 2 & 3 \\ 5 & 1 \end{bmatrix} \Rightarrow A^T = \begin{bmatrix} 2 & 5 \\ 3 & 1 \end{bmatrix}, \quad C^T = \begin{bmatrix} 2 \\ 1 \end{bmatrix}$$

$$CA = \begin{bmatrix} 9 & 7 \end{bmatrix}$$

$$\bar{O} = \begin{bmatrix} 2 & 1 \\ 9 & 7 \end{bmatrix}^T$$

$$\bar{O} = \begin{bmatrix} 2 & 9 \\ 1 & 7 \end{bmatrix}$$

$$\Rightarrow ATCT = \begin{bmatrix} 2 & 5 \\ 3 & 1 \end{bmatrix} \begin{bmatrix} 2 \\ 1 \end{bmatrix} = \begin{bmatrix} 9 \\ 7 \end{bmatrix}$$

Both are the same

$$\text{Eg. } \dot{x} = \begin{bmatrix} 1 & 0 \\ 0 & -1 \end{bmatrix}x + \begin{bmatrix} 1 \\ 0 \end{bmatrix}u, \quad x \in \mathbb{R}.$$

Find controllability.

$$\bar{C} = [B \ AB] = \begin{bmatrix} 1 & 1 \\ 0 & 0 \end{bmatrix}$$

$$\det(\bar{C}) = 0 \quad [\text{There is atleast one eigen-value } = 0]$$

$$\text{rank}(\bar{C}) = 1 \rightarrow \text{Rank-deficient matrix.}$$

controllability is not there.

$|C|=0 \Rightarrow$ Linear dependency on rows/columns \Rightarrow At least one $\lambda=0$.
 \Leftrightarrow Determinant is zero \Leftrightarrow Rank-deficient matrix.
 \rightarrow If one of the row/column is zero, it always becomes a rank-deficient matrix.
 \Rightarrow Not controllable.

MATLAB

$$\dot{x} = [1 \ 0]x + [1]u \rightarrow \text{find transfer function in MATLAB.}$$

$[ss\ 2\ tf]$

$$y = [1 \ 0]x$$

$\rightarrow 1$ and -1 are eigen values.

Eg $\dot{x} = \begin{bmatrix} 1 & 1 \\ -2 & -1 \end{bmatrix}x + \begin{bmatrix} 0 \\ 1 \end{bmatrix}u$. find observability and controllability.

$$y = [1 \ 0]x$$

$$\bar{\Omega} = [C^T \ A^T C^T] = \begin{bmatrix} 0 & [1 \ -2] \\ 1 & [1 \ -1] \end{bmatrix} \begin{bmatrix} 1 \\ 0 \end{bmatrix} = \begin{bmatrix} 1 & 1 \\ 0 & 1 \end{bmatrix}$$

$|\bar{\Omega}| = 1 \Rightarrow$ observable.

$$\left| \begin{array}{l} \det(\bar{\Omega}) = 1 \\ \text{Rank } (\bar{\Omega}) = 2 \end{array} \right.$$

$$\bar{\Omega} = [B^T \ AB^T] = \begin{bmatrix} 0 & [-1 \ 1] \\ 1 & [-2 \ -1] \end{bmatrix} \begin{bmatrix} 0 \\ 1 \end{bmatrix} = \begin{bmatrix} 0 & 1 \\ 1 & -1 \end{bmatrix}$$

$|\bar{\Omega}| \neq 0 \Rightarrow$ controllable.

$$\left| \begin{array}{l} \det(\bar{\Omega}) = 1 \\ \text{Rank } (\bar{\Omega}) = 2 \end{array} \right.$$

\rightarrow Both controllable and observable system \Leftrightarrow Minimum Realization of the system

\hookrightarrow No pole-zero cancellation.

\rightarrow Practical control systems are designed to be minimal realization system.

Let $A = \begin{bmatrix} 1 & 0 \\ 0 & -1 \end{bmatrix} \Rightarrow$ spectrum = {1, -1} \rightarrow eigen values value should be -ve;

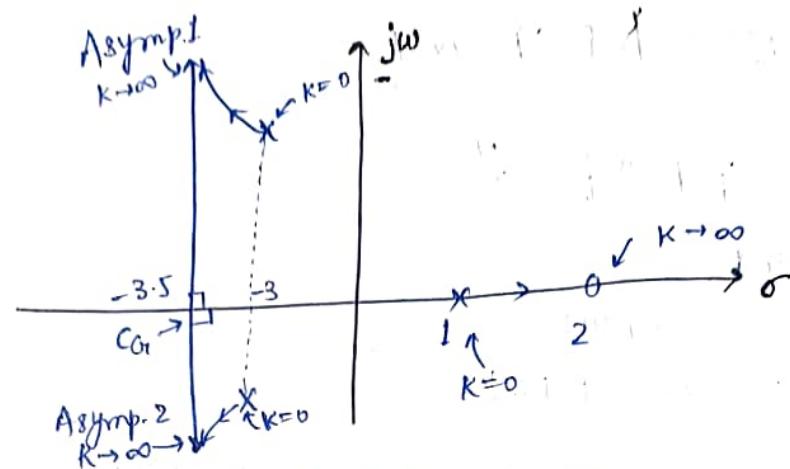
$B = \begin{bmatrix} 1 \\ 0 \end{bmatrix}$ \Rightarrow w unstable sys.

\rightarrow system is not controllable, but stabilizable.

\rightarrow stabilizable is a subset of controllable.

\hookrightarrow we can put an actuator at that position to make the sys. stable, still system might be uncontrollable.

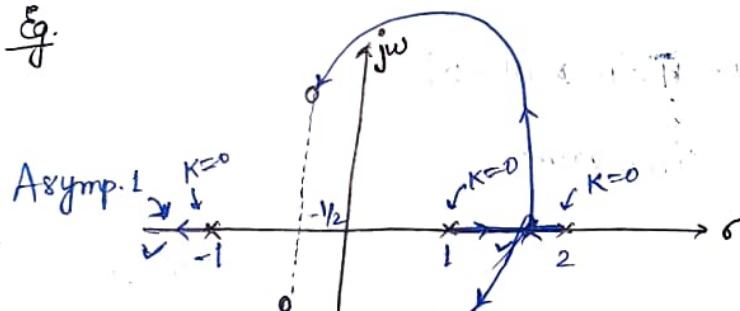
Ex.



$$\text{Centroid} = \frac{-3 - 3 + 1 - 2}{2} = -3.5$$

$$\angle_{\text{asy}} = \pm \frac{180^\circ (2k+1)}{2} = \pm 90^\circ$$

Ex.

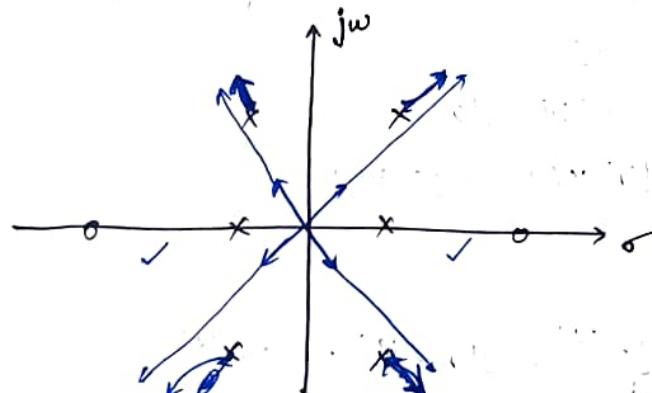


Draw the root locus.

$$\angle_{\text{asy}} = \pm \frac{180^\circ (2k+1)}{n-m}$$

$$\begin{aligned} C_G &= \frac{-1 + 1 + 2 + \frac{1}{2} + \frac{1}{2}}{1} \\ &= 3 \end{aligned}$$

Ex.



$$n-m = 4 \text{ (asymptotes)}$$

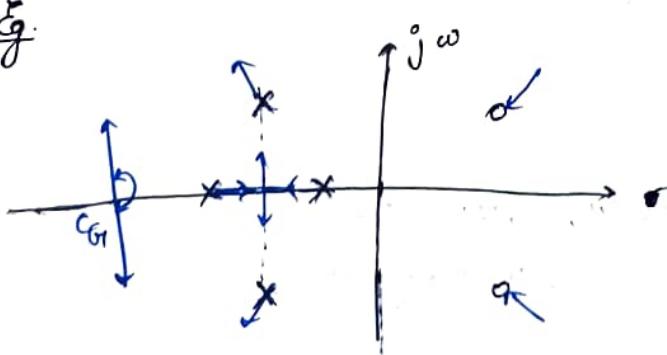
$$\pm \frac{180^\circ (2k+1)}{(n-m)}, k=0, 1, 2, \dots$$

$$= \pm 45^\circ (2 \times 0 + 1), = \pm 45^\circ$$

$$\pm 45^\circ (2 \times 1 + 1) = \pm 135^\circ$$

$$C_G = 0$$

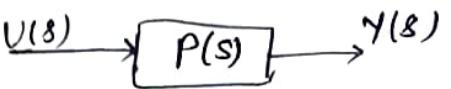
Ex.



$$n-m = 2 \text{ asymptotes}$$

$$\pm \frac{180^\circ (2k+1)}{2} = \pm 90^\circ$$

FREQUENCY RESPONSE



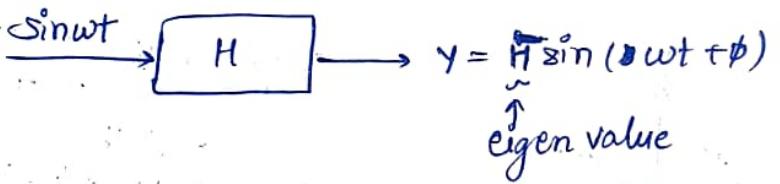
LTI stable $P(s)$

$$V(s) = V_0 \sin \omega t$$

$$Y(s) = A \sin(\omega t + \phi)$$

$e^{j\omega t} = \cos \omega t + j \sin \omega t$: eigen fn of LTI system

$Ax = \lambda x$, x : eigen vector
 λ : eigen value



03-10-2025

Eg. Given TF, $\frac{X(s)}{F(s)} = \frac{15}{(s+10)(s+11)}$. find the corresponding differential eqns. Also what is the order of the system.

Soln: Given, $\frac{X(s)}{F(s)} = \frac{15}{s^2 + 21s + 110}$

$$\Rightarrow (s^2 + 21s + 110) X(s) = 15 F(s)$$

Taking ILT,

$$\frac{d^2 x(t)}{dt^2} + 21 \frac{dx(t)}{dt} + 110 x(t) = 15 f(t) ..$$

$$\Rightarrow \ddot{x}(t) + 21 \dot{x}(t) + 110 x(t) = 15 f(t) \dots ①$$

Let $x_1(t) \triangleq x(t)$

$$x_2(t) \triangleq \dot{x}(t)$$

$$① \Rightarrow \ddot{x}_2(t) + 21 \dot{x}_2(t) + 110 x_1(t) = 15 f(t)$$

$$\Rightarrow \ddot{x}_2(t) = -110 x_1(t) - 21 x_2(t) + 15 f(t)$$

$$\dot{x}_1(t) = 0 \cdot x_1(t) + 1 \cdot x_2(t)$$

$$\dot{x} \triangleq \begin{bmatrix} \dot{x}_1(t) \\ \dot{x}_2(t) \end{bmatrix} = \begin{bmatrix} 0 & 1 \\ -110 & -21 \end{bmatrix} x(t) + \begin{bmatrix} 0 \\ 15 \end{bmatrix} f(t).$$

$$y(t) = [1 \ 0] x(t) = [1 \ 0] \begin{bmatrix} x_1(t) \\ x_2(t) \end{bmatrix}$$

$$\bar{C} = [B \ AB \ \dots \ A^{n-1}B]$$

$$= \begin{bmatrix} 0 & \begin{bmatrix} 0 & 1 \\ -110 & -21 \end{bmatrix} \begin{bmatrix} 0 \\ 15 \end{bmatrix} \end{bmatrix}$$

$$= \begin{bmatrix} 0 & 15 \\ 15 & -35 \end{bmatrix}$$

$|\bar{C}| \neq 0 \Rightarrow$ controllable.

$$|\bar{D}| = [C^T \ A^T C^T \ \dots] = \begin{bmatrix} 1 & \begin{bmatrix} 0 & -110 \\ 0 & 1 \end{bmatrix} \begin{bmatrix} 1 \\ 0 \end{bmatrix} \\ 0 & \begin{bmatrix} 1 & -21 \end{bmatrix} \begin{bmatrix} 0 \\ 15 \end{bmatrix} \end{bmatrix} = \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix}$$

$\Rightarrow |\bar{D}| \neq 0 \Rightarrow$ observable.

identity
(not singular)

Eg. Consider a unity (-ve) f/b system where plant TF is $\frac{4s+2}{s^2-s}$. Find if $s=-2$ is a closed loop solution/root of the system or not.

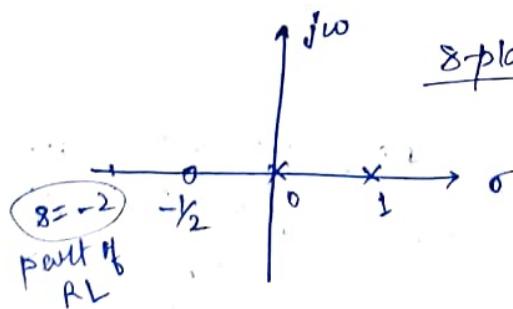
Soln: $G(s)H(s) = \frac{4s+2}{s(s+1)}$

$$1 + G(s)H(s) = 0 \Rightarrow G(s)H(s) = -1$$

$$\Rightarrow |G(s)H(s)| = 1$$

$$\angle G(s)H(s) = \pm 180^\circ (2k+1), \quad k=0, 1, \dots$$

$$\left| \begin{array}{l} \frac{GH}{1+GH} \\ \text{L.Pole:} \\ 1+GH=0 \end{array} \right.$$



s-plane

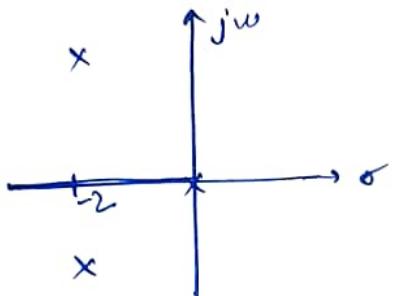
$$\left| \begin{array}{l} G(s)H(s)|_{-2} = \frac{4(-2)+2}{(-2)(-2-1)} = -1 \\ 1 + G(s)H(s) = 0 \end{array} \right.$$

Ex Consider OLTF with ^{unity} F/B be $G(s) = \frac{K}{s^3 + 4s^2 + 8s}$, $K \geq 0$.

- How many branches does the CL system have in RL?
- Find the region of the real axis that lies on RL.

Soln: $s^3 + 4s^2 + 8s = 0$
 $\Rightarrow s(s^2 + 4s + 8) = 0$
 $\Rightarrow s=0, -2 \pm j2$

Ans: 3
 (3 poles)

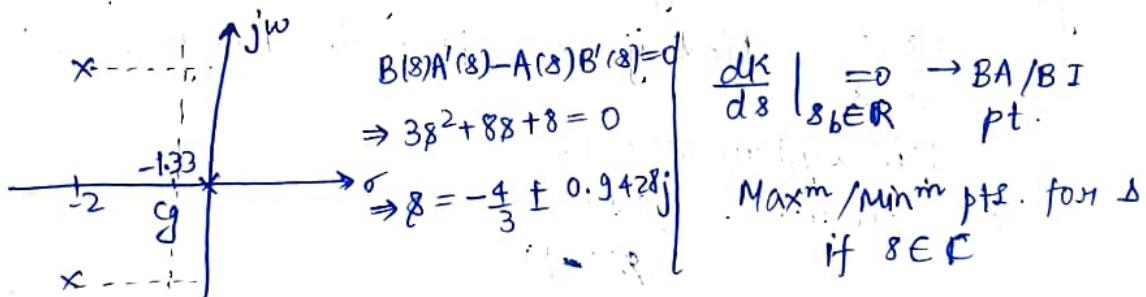


Ans: $(0, -\infty)$.

- Find if there're breakaway and BI points in RL.

Soln: Asymptote = $\pm \frac{180^\circ (2K+1)}{3} = \pm 60^\circ, \pm 180^\circ$

$$G = \frac{0 + (-2) + (-2)}{3} = -1.33$$



- find the range of K-values for which the CL system is stable.

[Using R-H / crossover pt.]

Soln: $1 + \frac{K}{s^3 + 4s^2 + 8s} = 0 \Rightarrow s^3 + 4s^2 + 8s + K = 0$
 $s = j\omega \Rightarrow -\omega^3 - 4\omega^2 + 8j\omega + K = 0$
 $\Rightarrow -\omega^3 + 8\omega = 0 \quad | \quad K = 4\omega^2$
 $\Rightarrow \omega = 0, \pm 2\sqrt{2}$
 If $\omega = 0 \Rightarrow K = 0$
 If $\omega = \pm 2\sqrt{2} \Rightarrow K = 32$

- Pole placement (controller design) \rightarrow controllability
- observable design \rightarrow observability

frequency domain analysis

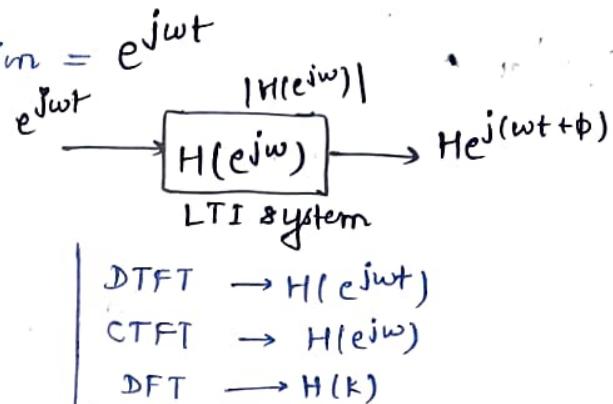
LTI system : eigen-function = $e^{j\omega t}$

ϕ : phase shift

A : magnitude / gain

① Magnitude

② Phase

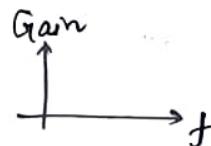


How can we visualise $H(j\omega)$?

- Bode plot
- Nyquist plot

① Bode plot

@Gain v.s. frequency



semilog : $y = \log \dots$
 x -linear
 }
 equal emphasis on
 low & high freq.
 \hookrightarrow but zero freq.
 cannot be
 represented.

② Nyquist plot

$\operatorname{Re}\{H(j\omega)\}$ v.s. $\operatorname{Im}\{H(j\omega)\}$

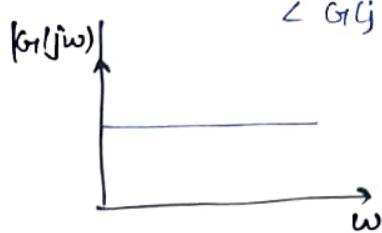
in the complex plane.

Constituent element / block:

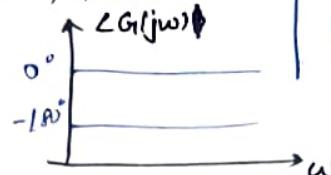
$$K, \frac{1}{s}, \delta, \frac{1}{1+sT}, T\delta + 1, \frac{\omega_n^2}{s^2 + 2\xi\omega_n s + \omega_n^2}, \frac{s^2 + 2\xi\omega_n s + \omega_n^2}{\omega_n^2}$$

Eg: $G_1(s) = K, K \in \mathbb{R}$

$$G_1(j\omega) = k, |G_1(j\omega)| (\text{dB}) = 20 \log_{10} |k| \text{ dB}$$



$$\angle G_1(j\omega) = \begin{cases} 0^\circ, & K > 0 \\ -180^\circ, & K < 0 \end{cases}$$



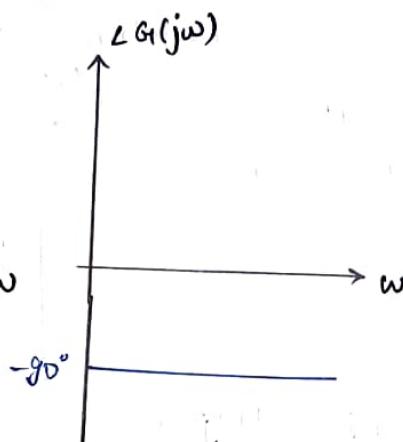
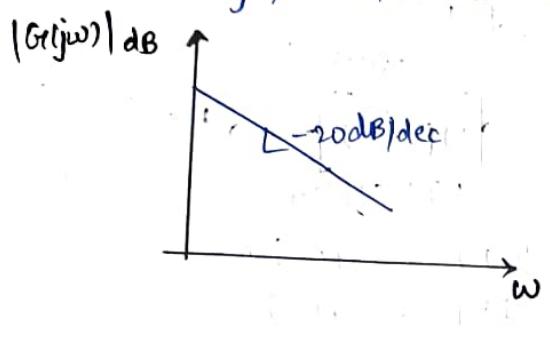
Decade \rightarrow 10 fold change
 in freq.
 Octave \rightarrow 2 fold change
 in freq.

$$\text{Eq. } G(s) = \frac{1}{s}$$

$$G(j\omega) = \frac{1}{j\omega}$$

$$|G(j\omega)| = \frac{1}{\omega}$$

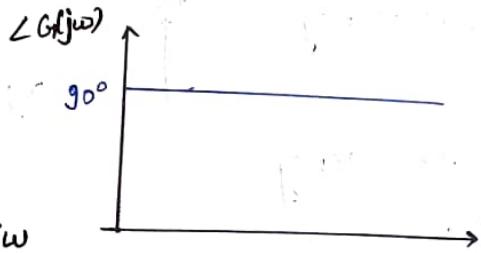
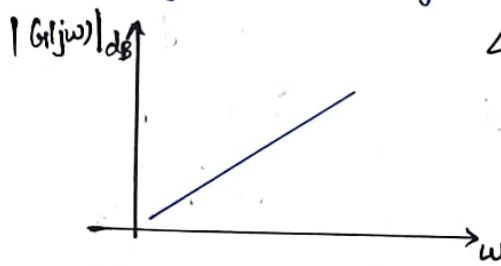
$$\angle G(j\omega) = -\tan^{-1}\infty = -90^\circ$$



$$\text{Eq. } G(s) = s$$

$$G(j\omega) = j\omega$$

$$|G(j\omega)| = \omega, \quad \angle G(j\omega) = 90^\circ$$



$$\text{At } \omega, |G(j\omega)| = 20 \log_{10} \left(\frac{1}{\omega} \right)$$

$$\begin{aligned} 10\omega, |G(j\omega)| &= 20 \log_{10} \left(\frac{1}{10\omega} \right) \\ &= -20 - 20 \log_{10} \omega \end{aligned}$$

$$\begin{aligned} 2\omega, |G(j\omega)| &= -20 \log (2\omega) \\ &= -20 \log 2 - 20 \log_{10} \omega \\ &= -E - 20 \log_{10} \omega \end{aligned}$$

$$\text{Eq. } G(s) = \frac{1}{1+sT}, \quad T > 0$$

$$G(j\omega) = \frac{1}{1+j\omega T}$$

$$|G(j\omega)| = \frac{1}{\sqrt{1+\omega^2 T^2}}$$

$$\angle G(j\omega) = -\tan^{-1}(\omega T)$$

At $\omega \ll \frac{1}{T}$,

$|G(j\omega)| = 1$, $|G(j\omega)|_{dB} = 0$
(low-frequency asymptote)

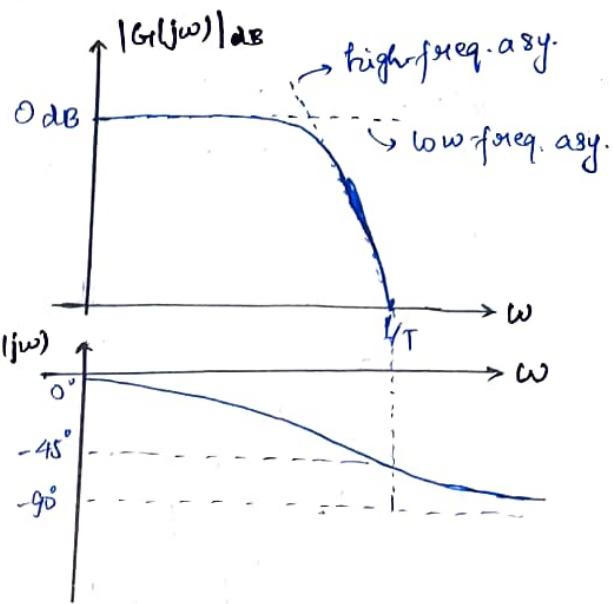
At $\omega > \frac{1}{T}$,

$$|G(j\omega)| \approx \frac{1}{\omega T}$$

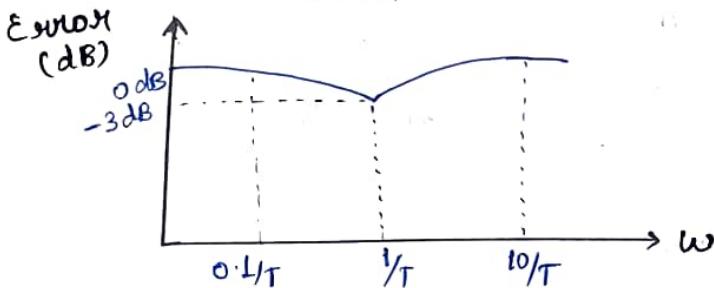
$$|G(j\omega)|_{dB} = -20 \log \omega T \text{ dB}$$

(High-frequency asymptote)

↪ Asymptotic approach

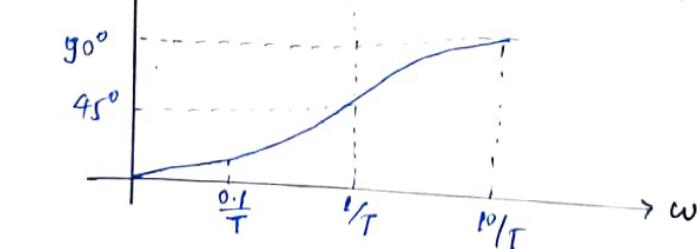
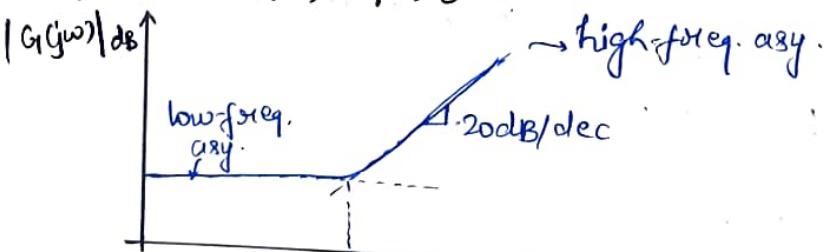


ω	$ G(j\omega) _{dB}$	Asymptotic value (dB)
$1/T$	-3.01	0
$2/T$	-6.99	-6.02
$0.5/T$	-0.97	0
$10/T$	-20.04	-20
$0.1/T$	-0.04	0



$\omega = 1/T$: corner freq.,
3-dB cutoff freq.

Eg. $G(s) = 1 + sT$, $T > 0$



$$\text{Eq } G(s) = \frac{\omega_n^2}{s^2 + 2\xi\omega_n s + \omega_n^2}$$

$$G(s) = \frac{1}{\frac{s^2}{\omega_n^2} + \frac{2\xi s}{\omega_n} + 1}$$

$$G(j\omega) = \frac{1}{1 - \frac{\omega^2}{\omega_n^2} + 2j\frac{\xi\omega}{\omega_n}}$$

$$|G(j\omega)| = \frac{1}{\sqrt{\left(1 - \left(\frac{\omega}{\omega_n}\right)^2\right)^2 + \left(\frac{2\xi\omega}{\omega_n}\right)^2}}$$

$$\angle G(j\omega) = -\tan^{-1} \left(\frac{\frac{2\xi\omega}{\omega_n}}{1 - \left(\frac{\omega}{\omega_n}\right)^2} \right)$$

$$|G(j\omega)|_{\text{dB}} = -20 \log_{10} \sqrt{\left(1 - \left(\frac{\omega}{\omega_n}\right)^2\right)^2 + \left(\frac{2\xi\omega}{\omega_n}\right)^2} \text{ dB}$$

If $\omega \ll \omega_n$, $|G(j\omega)|_{\text{dB}} \approx 0 \text{ dB}$ (low-freq. asy.)

$\omega \gg \omega_n$, $|G(j\omega)|_{\text{dB}} = -40 \log \left(\frac{\omega}{\omega_n} \right) \text{ dB}$ (high-freq. asy.)

At $\omega = \omega_n$,

$$|G(j\omega)|_{\text{dB}} = -20 \log_{10} 2\xi$$

under-damped system, $\xi = 0.5$:

$$|G(j\omega)|_{\text{dB}} \approx 0 \text{ dB.}$$

Ex. Find the freq. at which $|G(j\omega)|$ is max.

$$|G(j\omega)| = \frac{1}{\sqrt{\left(1 - \left(\frac{\omega}{\omega_n}\right)^2\right)^2 + \left(\frac{2\xi\omega}{\omega_n}\right)^2}}$$

$$\text{let } g(\omega) = \left(1 - \left(\frac{\omega}{\omega_n}\right)^2\right)^2 + \left(\frac{2\xi\omega}{\omega_n}\right)^2$$

$$\therefore g'(\omega) = 0$$

$$\Rightarrow \omega = 0 \text{ or } \omega_n \sqrt{1 - 2\xi^2}$$

$$g'(w) = 2 \left(1 - \left(\frac{w}{\omega_n} \right)^2 \right) \left(-\frac{2w}{\omega_n} \right) \left(\frac{1}{\omega_n} \right) + 2 \left(2\zeta \frac{w}{\omega_n} \right) \left(\frac{2\zeta}{\omega_n} \right)$$

$$\begin{aligned} g'(w) = 0 &\Rightarrow \left[2\zeta^2 - \left(1 - \frac{w^2}{\omega_n^2} \right) \right] \frac{w}{\omega_n^2} = 0 \\ &\Rightarrow \left[2\zeta^2 - \left(1 - \frac{w^2}{\omega_n^2} \right) \right] w = 0 \quad [\because \omega_n \neq 0] \end{aligned}$$

Either, $w=0$

$$\text{or, } 2\zeta^2 = 1 - \frac{w^2}{\omega_n^2}$$

$$\omega_R = \omega_n \sqrt{1 - 2\zeta^2} \quad (\text{Resonance})$$

ω_R exist if $0 < \zeta < 1/\sqrt{2}$.

$$\omega_d = \omega_n \sqrt{1 - \zeta^2} \quad (\text{damped})$$

$$\therefore \omega_d > \omega_R$$

Resonance freq.: Freq. of max gain of T.F.

Minimum and Non-Minimum Phase System

Consider systems: $G_1 = \frac{1+sT}{1+sT_1}$, $T, T_1 > 0 \rightarrow \text{MP, stable}$

$G_2 = \frac{1-sT}{1+sT_1}$, $T, T_1 > 0 \rightarrow \text{NMP, stable}$

Frequency response of the systems,

$$G_1(jw) = \frac{1+j(Tw)}{1+j(T_1w)}$$

$$G_2(jw) = \frac{1-j(Tw)}{1+j(T_1w)}$$

$$\Rightarrow |G_1(jw)| = |G_2(jw)| = \sqrt{\frac{1+w^2T^2}{1+w^2T_1^2}}$$

$$\angle G_1(jw) = \tan^{-1}(wT) - \tan^{-1}(wT_1)$$

$$\angle G_2(jw) = -\tan^{-1}(wT) - \tan^{-1}(wT_1)$$

$$w=0 \Rightarrow \angle G_1 = \angle G_2 = 0$$

$$w \rightarrow \infty \Rightarrow \angle G_1 = 0^\circ, \angle G_2 = -180^\circ \rightarrow \text{NMP system}$$

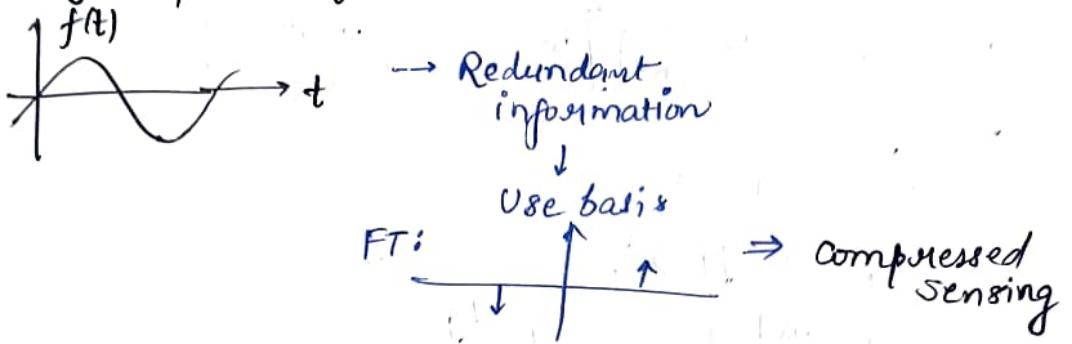
↪ Same mag. response, diff. phase response.

Hyper-imaginary no.
 j, j_1, j_2
(different axes)

first go in wrong dirn
then gets rectified

$$\text{Qg: } G_1(s) = \frac{s-2}{s+4}, \quad G_2(s) = \frac{s+2}{s+4}.$$

Frequency Response of Undamped Systems



Need not satisfy: $f_s \geq 2f_m$
 [Nyquist-Shannon Theorem]
 $f_s \leq 2f_m$

$\lambda(A) = 0 \rightarrow$ Redundant rows & columns

$\xi = 0$ (Undamped System)

$$G(s) = \frac{\omega_n^2}{s^2 + \omega_n^2} \Rightarrow G(j\omega) = \frac{\omega_n^2}{(j\omega)^2 + \omega_n^2} = \frac{1}{1 - \omega^2/\omega_n^2}$$

$$\Rightarrow |G(j\omega)| = \frac{1}{\left|1 - \frac{\omega^2}{\omega_n^2}\right|}$$

$\omega \rightarrow \omega_n \Rightarrow |G(j\omega)| \rightarrow \infty \Rightarrow$ Don't take freq. as natural freq.

Bode Plot : [Fourier Transform]

$$\text{Ex: } G(s) = \frac{s}{(s+1)(s+10)}$$

Convert into time constant form.

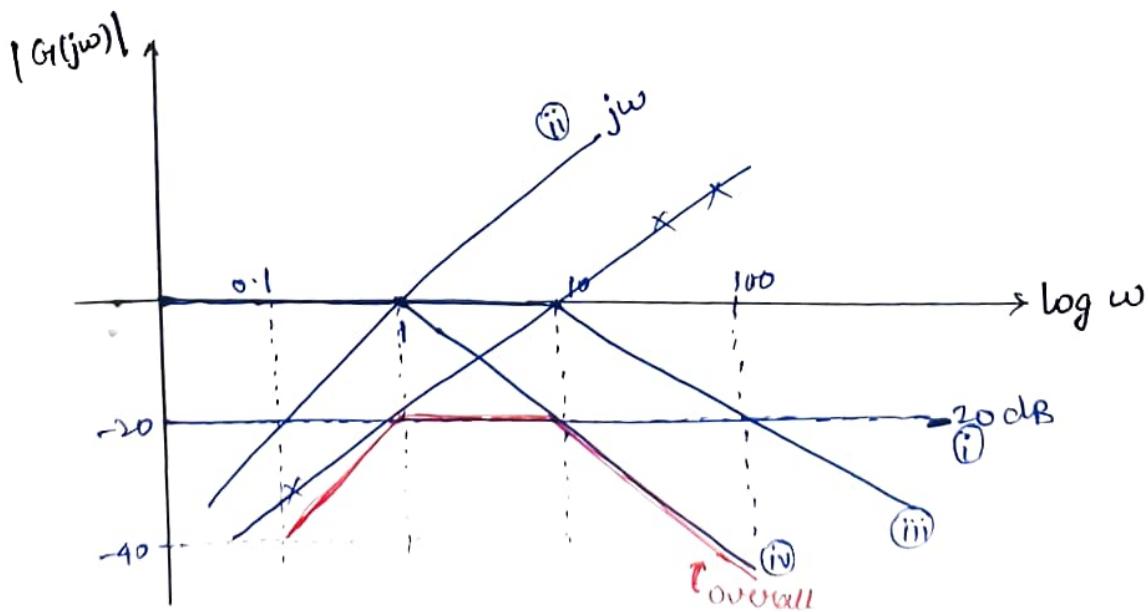
$$G(s) = \frac{s/10}{(1+s)(1+s/10)}$$

$$G(j\omega) = \frac{j\omega/10}{(1+j\omega)(1+j\omega/10)} = \frac{j\omega/10}{1 - \frac{\omega^2}{10} + \frac{11}{10}j\omega}$$

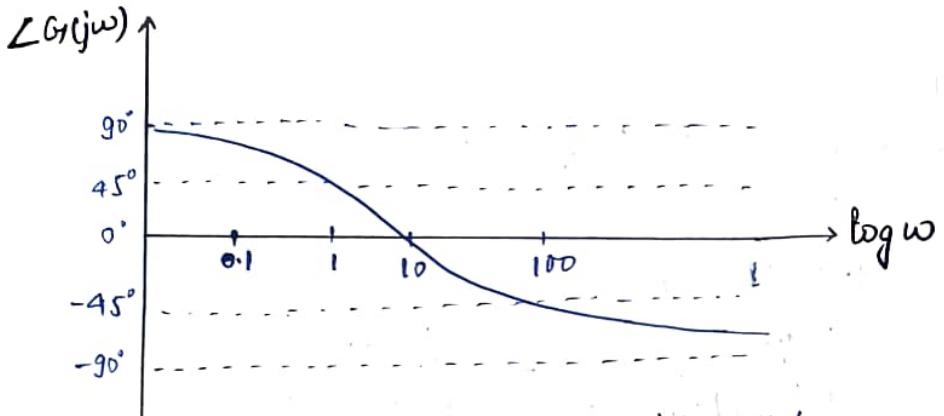
$$|G(j\omega)| = \frac{\omega/10}{\sqrt{(1 - \frac{\omega^2}{10})^2 + (\frac{11}{10}\omega)^2}}$$

$$G(j\omega) = (0.1)(j\omega) \left(\frac{1}{1+j\omega} \right) \left(\frac{1}{1+0.1j\omega} \right)$$

- (i) 0.1 : $20 \log(0.1) = -20 \text{ dB}$ / No slope
- (ii) $j\omega$: $20 \log(\omega) = +20 \text{ dB/dec}$
- (iii) $\frac{1}{1+j\omega/10}$: $-20 \log \sqrt{1 + \left(\frac{\omega}{10}\right)^2}$ / -20 dB/dec
- (iv) $\frac{1}{1+j\omega}$: $-20 \log \sqrt{1-\omega^2}$ / -20 dB/dec.



$$\angle G_1(j\omega) = 0 + 90^\circ - \tan^{-1}(\omega) - \tan^{-1}(0.1\omega)$$



→ For minimum phase, $\angle G_1(j\omega) = 90^\circ(n-m) - \text{poles} + \text{zeros}$

Lead and Lag Compensators.

$$G(s) = \frac{s+z}{s+p}$$

$$\angle G(s) = \angle s+z - \angle s+p$$

↗ +ve: ϕ -lead : Lead compensator
↘ -ve: ϕ -lag : lag compensator

$$G_C(s) = \frac{s + \frac{1}{\alpha T}}{s + \frac{1}{\beta T}} \Rightarrow G_C(j\omega) = \frac{j\omega + \frac{1}{\alpha T}}{j\omega + \frac{1}{\beta T}}$$

$$\Rightarrow \angle G_C(j\omega) = \tan^{-1}(\alpha \omega T) - \tan^{-1}(\beta \omega T) > 0$$

$$\alpha = 1, \beta = 2$$

$$\Rightarrow \angle G_C(j\omega) = \tan^{-1}(\omega T) - \tan^{-1}(2\omega T) < 0$$

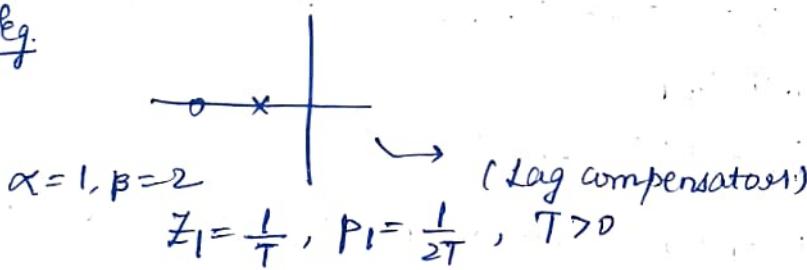
If $\alpha < \beta \Rightarrow$ Lag compensator.

$$\alpha = 2, \beta = 1$$

$$\Rightarrow \angle G_C(j\omega) = \tan^{-1}(2\omega T) - \tan^{-1}(\omega T) > 0^\circ$$

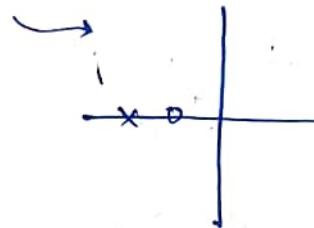
If $\alpha > \beta \Rightarrow$ Lead compensator.

Eg.



Eg. $\alpha = 2, \beta = 1$ (Lead)

$$Z_1 = \frac{1}{2T}, P_1 = \frac{1}{T}$$

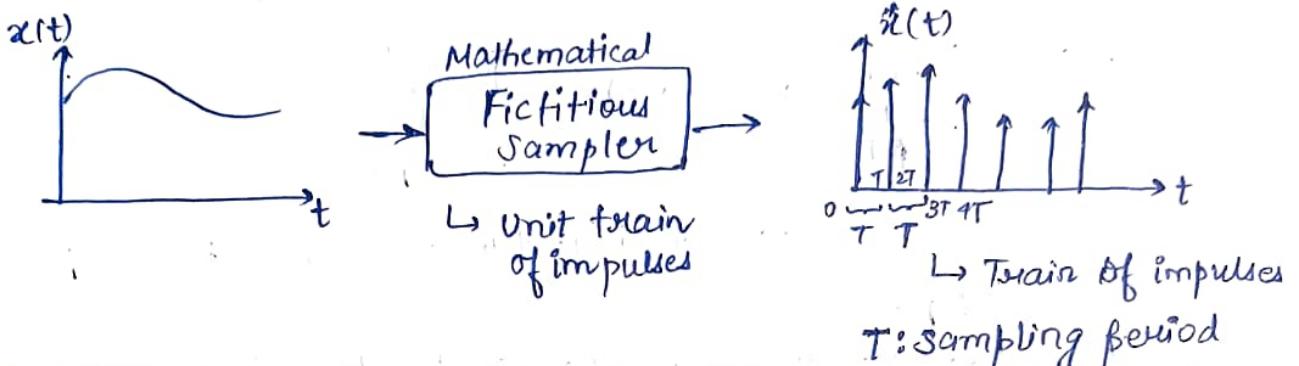


check how increasing GIM changes BW.

Discrete-Time Control System

$x(t)$: Continuous-time / analog signal

Sampling: $f_s \geq 2f_m$, f_m : max^m freq. content in signal.



Let $x^*(t)$ denote the impulse-sampled o/p in a sequence of impulses.

Train of impulses, $x^*(t)$, is given by

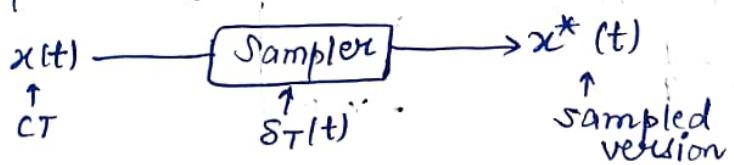
$$x^*(t) = \sum_{k=0}^{\infty} x(kT) \delta(t-kT) \quad \dots \textcircled{1}$$

$$x^*(t) = x(0) \delta(t) + x(T) \delta(t-T) + \dots + x(KT) \delta(t-KT) + \dots \quad \textcircled{2}$$

Define a train of unit impulses,

$$\delta_T(t) = \sum_{k=0}^{\infty} \delta(t-kT) \quad \dots \textcircled{3}$$

The sampler o/p is the product of the CT signal and the unit impulse sequence.



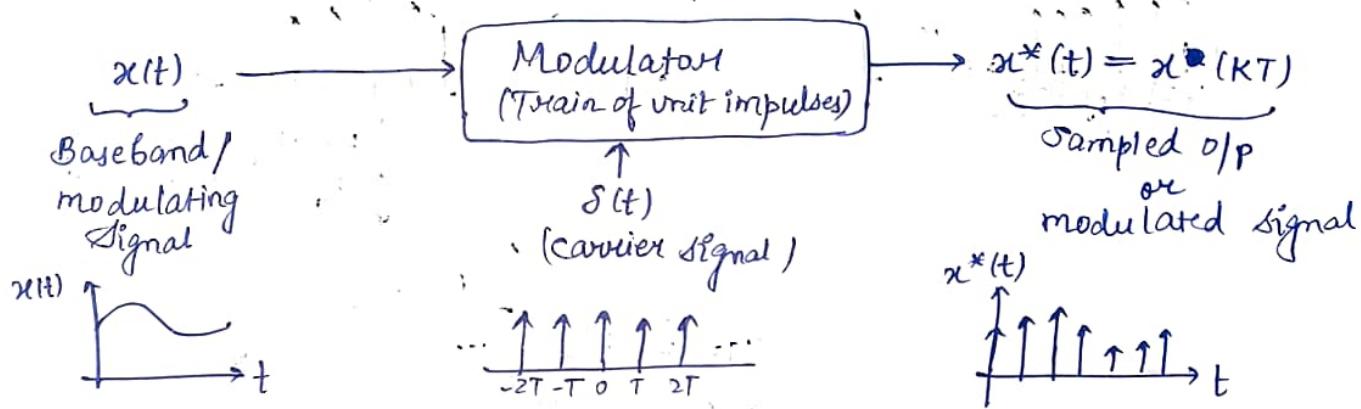
Consider the LT of $\textcircled{2}$.

$$\begin{aligned} x^*(s) &= \mathcal{L}[x^*(t)] \\ &= \mathcal{L}[x(0)\delta(t) + x(T)\delta(t-T) + \dots + x(KT)\delta(t-KT) + \dots] \\ &= x(0) + x(T)e^{-sT} + \dots + x(KT)e^{-sKT} + \dots \\ &= \sum_{k=0}^{\infty} x(kT) e^{-sKT} \end{aligned}$$

Consider $Z = e^{sT}$ or $s = \frac{1}{T} \ln(Z)$

$$x^*(s) \Big|_{s=\frac{1}{T} \ln(Z)} = \sum_{k=0}^{\infty} x(kT) Z^{-k}$$

$$= x(Z) \xrightarrow{Z} Z(x^*(t)) = Z(x^*(kT)).$$



Q1 For any LTI systems, apart from complex exponentials / sinusoids can there be any other basis functions?

↳ Yes, infinitely many.

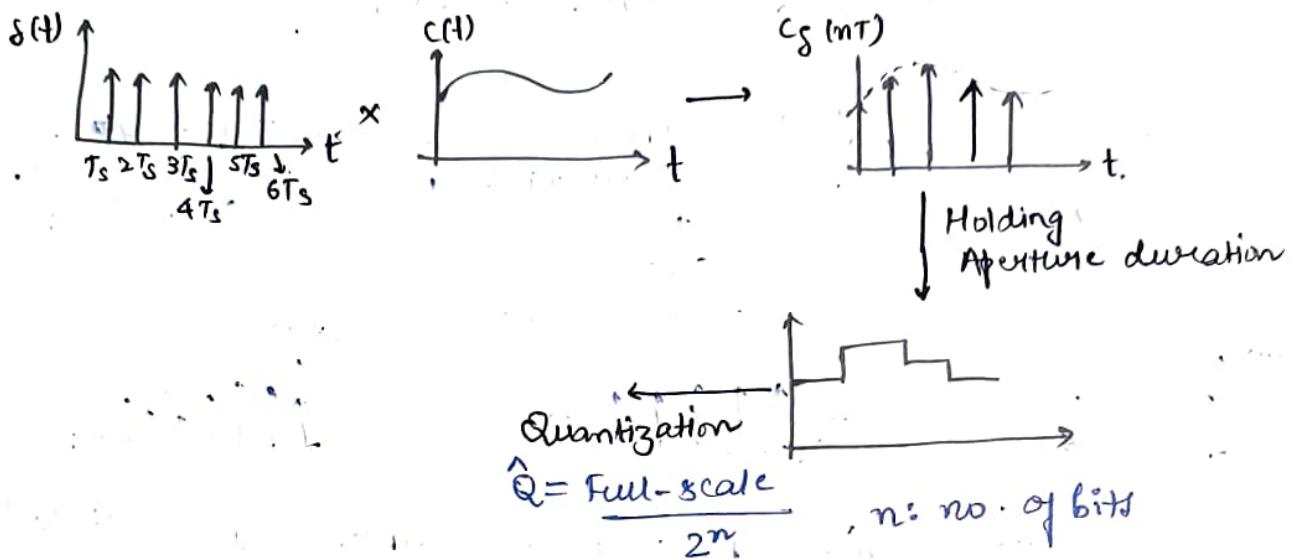
↳ But for LTI systems, only exponentials are the eigenfunctions.

Digital Control System:

S/H: Sample and Hold

ZOH: Zero order Hold

$e(t) \rightarrow \text{sample} \rightarrow \text{Holding} \rightarrow \text{ADC}$



ZOH (Zero Order Hold):

$$y(t) = x(n), \quad kT_s \leq n \leq (k+1)T_s, \quad k=0, 1, 2, 3, \dots$$

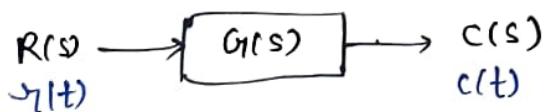
\uparrow
Hold ckt. eqn

- Federated learning
- Edge computing

07-11-2025

Derivation of Pulse Transfer Function

↪ LTI



$$G(s) = \frac{C(s)}{R(s)}, \text{ with ZIC.}$$

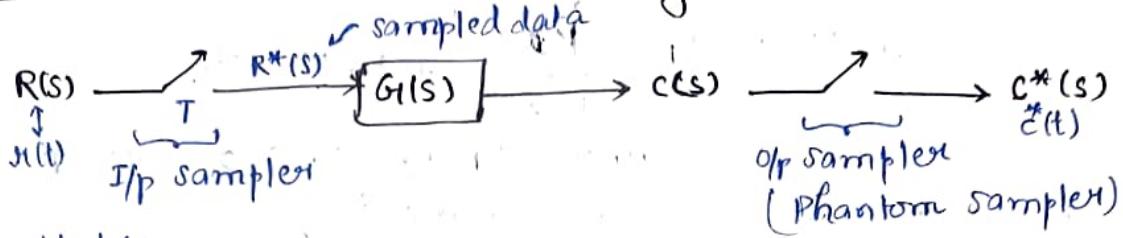
- Feedback control / ~~feedback~~ Feedforward control.
Unmodelled dynamics $\rightarrow f(x(t))$

$$\|f(x, t)\| < M < 0$$

↪ uncertainty, bounded disturbance

$$\hookrightarrow \|s(t)\| < N < \infty.$$

* OGATA: Discrete-Time control System [Book]



Sampled I/P,

$$r^*(t) = \sum_{k=0}^{\infty} r(kT) s(t - kT) \dots ①$$

$$c(t) = \sum_{k=0}^{\infty} r(kT) g(t - kT) \dots ②$$

We have,

$$c(z) = \sum_{n=0}^{\infty} c(nT) z^{-n} \dots ③$$

Put ③ in ② with $t = nT$.

$$\Rightarrow c(nT) = \sum_{k=0}^{\infty} r(kT) g(nT - kT) \dots ④$$

Put ④ in ③.

$$\Rightarrow c(z) = \sum_{n=0}^{\infty} \sum_{k=0}^{\infty} r(kT) g((n-k)T) z^{-n} \dots ⑤$$

Let $m \triangleq n - k$. Then,

$$c(z) = \sum_{m+k=0}^{\infty} \sum_{k=0}^{\infty} r(kT) g(mT) z^{-(im+k)}$$

$$= \sum_{m=0}^{\infty} g(mT) z^{-m} \sum_{k=0}^{\infty} r(kT) z^{-k}$$

$\left[\begin{array}{l} k \geq 0, m \leq 0 \Rightarrow m = 0 \\ g(mT) \text{ doesn't exist for } m < 0 \end{array} \right]$

$$\therefore c(z) = G(z) \cdot R(z)$$

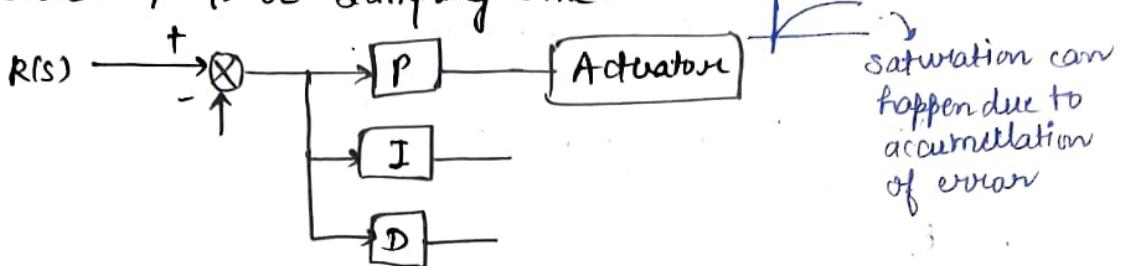
$\begin{matrix} \uparrow \text{Sampled O/P} & \uparrow \text{Pulse TF} & \uparrow \text{Sampling I/P} \\ zT & & zT \end{matrix}$

Prove that the pulse TF of a digital PID controller is

$$G_{PID}(z) = K_p + \frac{K_I}{1-z^{-1}} + K_D(1-z^{-1}),$$

where K_p, K_I & $K_D \geq 0$ are proportional, integral and derivative gain, respectively.

Consider 'T' to be sampling time.



PID controller : $K_p e(t) + K_I \int e(t) dt + K_D \frac{de(t)}{dt}$

- Makes system sluggish (lag also) ← integration
- error also accumulates
- anti windup design
- for steady state response
- lags the process
- can introduce damping
- for transient response design
- Noise can be amplified through derivative controller
- ↑ the BW
- speeds up the process

Realization of Digital Filter and Digital Controller

↳ Software OR Hardware

Computer

DSP processor

(Adder, multiplier, delay (shift register))

→ DC is a part of DF. (special type)

DF: A sophisticated algorithm that takes some sequence as i/p and provides o/p in a desired type of sequence.

→ The general form of pulse TF b/w o/p $Y(z)$ and i/p $X(z)$ is

$$G(z) = \frac{Y(z)}{X(z)} = \frac{b_0 + b_1 z^{-1} + b_2 z^{-2} + \dots + b_m z^{-m}}{1 + a_1 z^{-1} + a_2 z^{-2} + \dots + a_n z^{-n}}, \quad n \geq m$$

$$Y(z) = G(z) X(z)$$

where a_i and b_i 's are real coefficients.

- ① Direct Programming
- ② Standard Programming

a_i and b_i as multipliers in filter representation:

Direct structure

↳ Direct Form - I

↳ Direct Form - II /

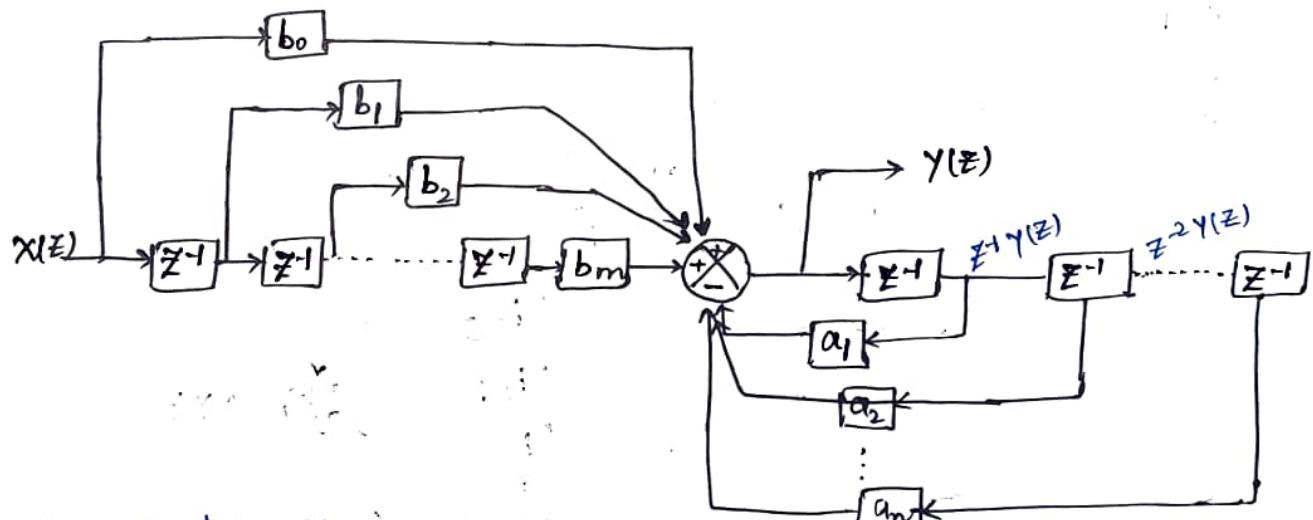
Canonical form /

Standard programming

Direct Programming (Form-I)

$$G(z) = \frac{Y(z)}{X(z)} = \frac{b_0 + b_1 z^{-1} + b_2 z^{-2} + \dots + b_m z^{-m}}{1 + a_1 z^{-1} + a_2 z^{-2} + \dots + a_n z^{-n}}$$

$$\begin{aligned} Y(z) = & -a_1 z^{-1} Y(z) - a_2 z^{-2} Y(z) - \dots - a_n z^{-n} Y(z) \\ & + b_0 X(z) + b_1 z^{-1} X(z) + \dots + b_m z^{-m} X(z) \end{aligned}$$



DF-I: $(n+m)$ delay blocks

DF-II: max (n, m) blocks.

(std./canonical)

Standard Programming / Canonical Structure / DF-II

$$\frac{Y(z)}{X(z)} = \frac{Y(z)}{H(z)} \cdot \frac{H(z)}{X(z)} = \frac{(b_0 + b_1 z^{-1} + b_2 z^{-2} + \dots + b_m z^{-m})}{(1 + a_1 z^{-1} + a_2 z^{-2} + \dots + a_n z^{-n})}$$

where

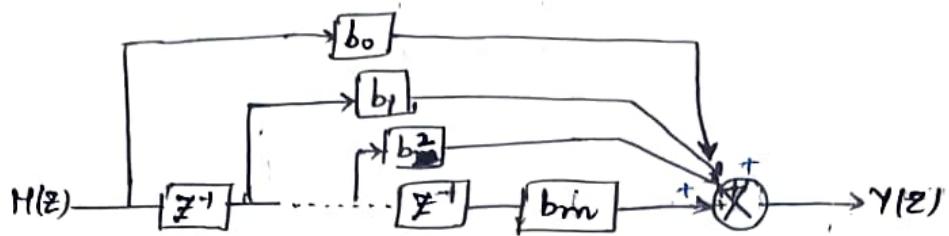
$$\frac{Y(z)}{H(z)} = b_0 + b_1 z^{-1} + b_2 z^{-2} + \dots + b_m z^{-m} \quad \dots \textcircled{1}$$

$$\frac{H(z)}{X(z)} = \frac{1}{1 + a_1 z^{-1} + a_2 z^{-2} + \dots + a_n z^{-n}} \quad \dots \textcircled{2}$$

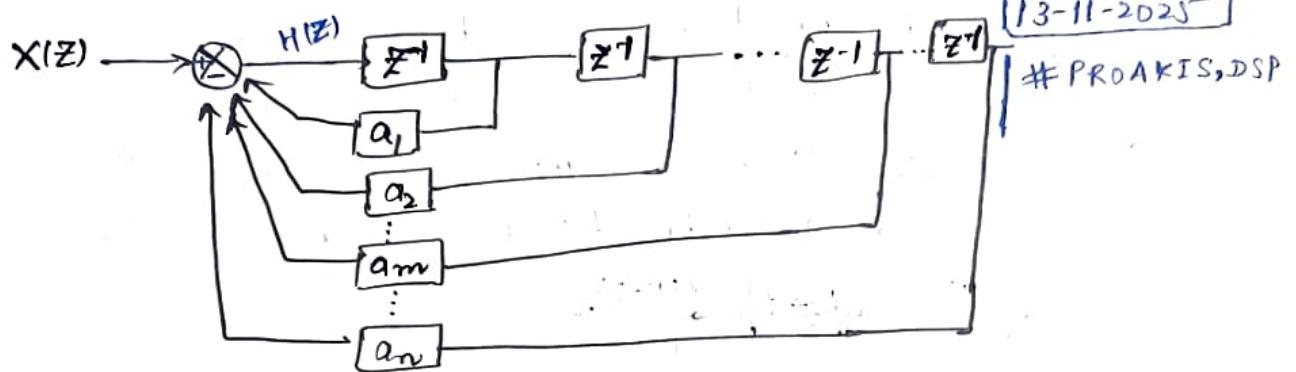
$$\textcircled{1} \Rightarrow Y(z) = b_0 H(z) + b_1 z^{-1} H(z) + b_2 z^{-2} H(z) + \dots + b_m z^{-m} H(z) \quad \dots \textcircled{3}$$

$$\textcircled{2} \Rightarrow H(z) = X(z) - a_1 z^{-1} H(z) - a_2 z^{-2} H(z) - \dots - a_n z^{-n} H(z) \quad \dots \textcircled{4}$$

\textcircled{3}:



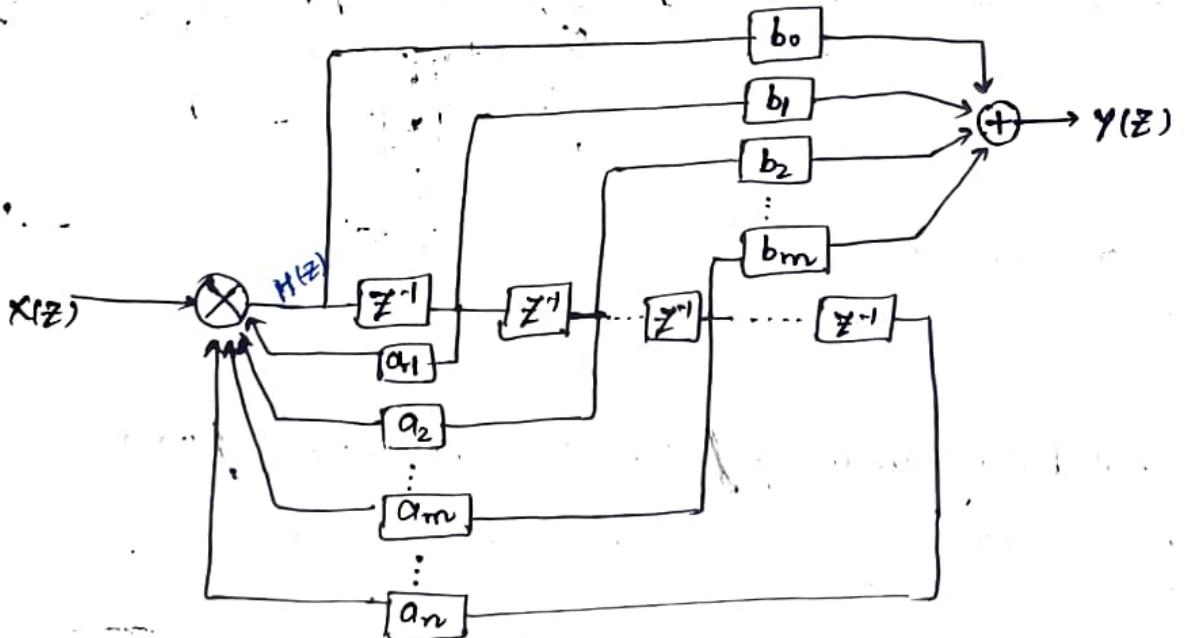
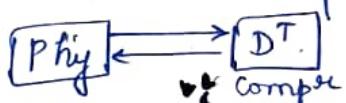
\textcircled{4}:



- * Pole Placement

- * Observer Design

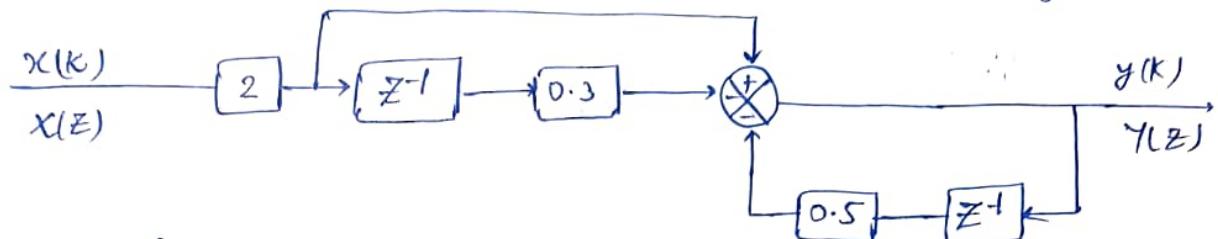
- * Digital twin : Real time replica of the system modelled.



Eg. Obtain the block diagrams in direct and standard program for the following pulse T.F:

$$\frac{Y(z)}{X(z)} = \frac{2 - 0.6z^{-1}}{1 - 0.5z^{-1}}$$

Soln: $Y(z) = -0.5z^{-1}Y(z) + 2X(z) - 0.6z^{-1}X(z)$ [Direct stand. Bwg. 7]

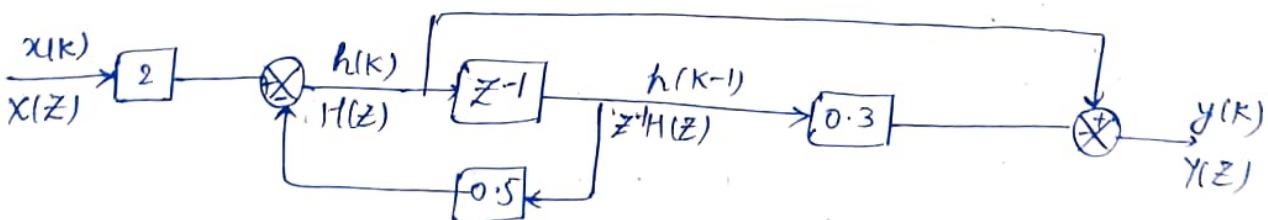
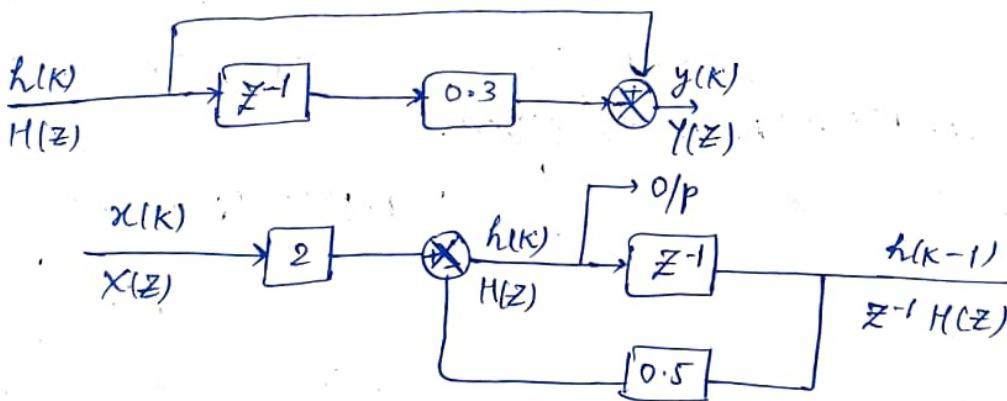


Standard Program :

$$\frac{Y(z)}{X(z)} = \frac{Y(z)}{H(z)} \cdot \frac{H(z)}{X(z)} = (1 - 0.3z^{-1}) \frac{2}{1 + 0.5z^{-1}}$$

where, $\frac{Y(z)}{H(z)} = (1 - 0.3z^{-1})$, $\frac{H(z)}{X(z)} = \frac{2}{1 + 0.5z^{-1}}$

$$\Rightarrow Y(z) = H(z) - 0.3z^{-1}H(z) \quad \Rightarrow H(z) = -0.5z^{-1}H(s) + 2X(z)$$



Sources of error in Digital Filters:

- ① Quantization error
- ② Accumulation error.

$$y(k) = \underbrace{2y(k-1)}_{\text{Quantization error}} + \underbrace{3y(k-2)}_{\text{Accumulation error}} + \dots + 2x(k) + 3x(k-1) + \dots$$

- ③ Coefficient Quantization error

$$\frac{Y(z)}{X(z)} = \frac{b_0 z^m + b_1 z^{m-1} + \dots}{1 + a_0 z^n + \dots}$$

IIR Filter \rightarrow Bilinear Transformation.

IIR / FIR

$$G(z) = \frac{Y(z)}{X(z)} = \frac{b_0 + b_1 z^{-1} + \dots + b_m z^{-m}}{1 + a_1 z^{-1} + a_2 z^{-2} + \dots + a_n z^{-n}}, \quad n > m \quad \dots \textcircled{1}$$

Difference eqn:

$$y(k) = -a_1 y(k-1) - a_2 y(k-2) - \dots - a_n y(k-n) \\ + b_0 x(k) + b_1 x(k-1) + \dots + b_m x(k-m) \quad \dots \textcircled{2}$$

If $\exists i \in \{1, 2, \dots, n\}$ such that $a_i \neq 0$, then the filter is called IIR Filter / Recursive filter.

If $\forall i, a_i = 0, i \in \{1, 2, \dots, n\} \Rightarrow$ FIR Filter / Non-Recursive filter.

Moving-average filter.

Let $g(kT)$ be the finite impulse / weighting sequence of the FIR filter.

If $x(kT)$ is the sequence applied to this filter, the o/p $y(kT)$

$$\text{i.e. } y(kT) = \sum_{h=0}^K g(hT) x(kT - hT),$$

i.e., o/p is convolution of i/p sequence and weighting seq.

$$= g(0)x(kT) + g(T)x((k-1)T) + \dots + g(kT)x(0) \quad \dots \textcircled{A}$$

When 'K' is large, we need to store more no. of past inputs.

⇒ Memory requirement becomes large.

- Instead we keep track of past 'N' i/p samples including the current sample.

Let's process 'N' immediate past i/p samples, $x((K-1)T)$, $x((K-2)T)$, ..., $x((K-N)T)$ and current sample $x(KT)$, i.e.,

$$y(KT) = g(0)x(KT) + g(T)x((K-1)T) + \dots + g(NT)x((K-N)T) \dots \textcircled{B}$$

$$\mathbb{Z}T \Rightarrow \boxed{Y(z) = g(0)x(z) + g(T)z^1x(z) + \dots + g(NT)z^{-N}x(z)}$$

↳ Moving Average/
Non-recursive.

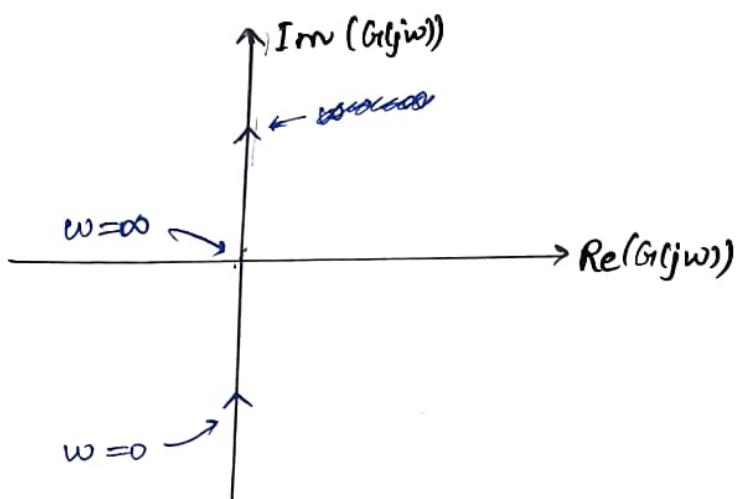
Nyquist Plot

↪ $\operatorname{Re}\{G(j\omega) H(j\omega)\}$ vs $\operatorname{Im}\{G(j\omega) H(j\omega)\}$ as ω is varied.

Eg. $G(s) = \frac{1}{s}$.

$$G(j\omega) = \frac{1}{j\omega} = \frac{j\omega}{j^2\omega} = 0 + j\left(-\frac{1}{\omega}\right) = \frac{1}{\omega} e^{-90^\circ}$$

ω	$\operatorname{Re}(G(j\omega))$	$\operatorname{Im}(G(j\omega))$	$ G(j\omega) $	$\angle G(j\omega)$
0	0	$-\infty$	∞	-90°
∞	0	0	0	-90°



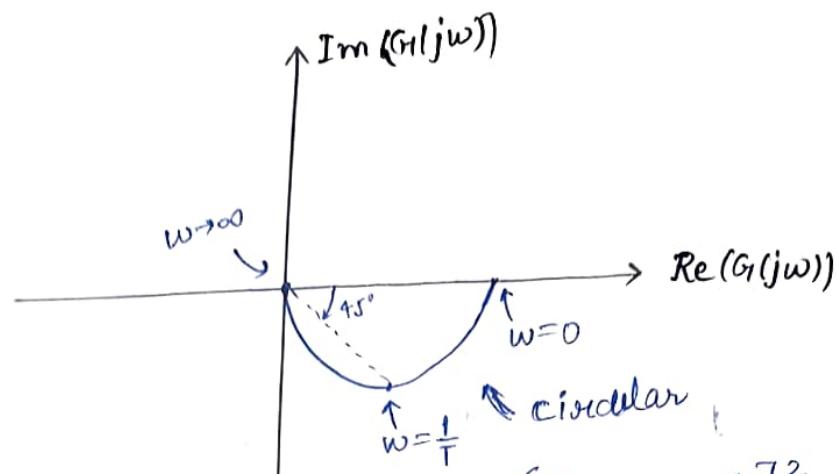
Eg. $G(s) = \frac{1}{1+sT}$

$$G(j\omega) = \frac{1}{1+j\omega T} = \frac{1-j\omega T}{1+\omega^2 T^2} = \frac{1}{1+\omega^2 T^2} - \frac{j\omega T}{1+\omega^2 T^2}$$

$$|G(j\omega)| = \frac{1}{\sqrt{1+\omega^2 T^2}}$$

$$\angle G(j\omega) = -\tan^{-1}(\omega T)$$

ω	$\operatorname{Re}(G(j\omega))$	$\operatorname{Im}(G(j\omega))$	$ G(j\omega) $	$\angle G(j\omega)$
0	1	0	1	0
$1/T$	$1/2$	$-1/2$	$1/\sqrt{2}$	-45°
∞	0	0	0	-90°



$$\left[\frac{1}{1+w^2T^2} - \frac{1}{2} \right]^2 + \left[\frac{-w^2}{1+w^2T^2} \right] = \left(\frac{1}{2} \right)^2$$

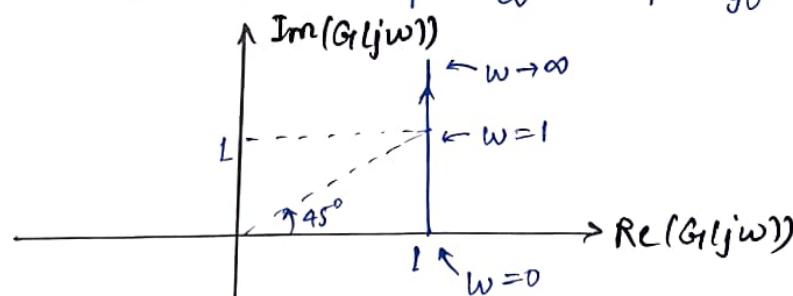
Eg. $G(s) = 1 + sT$

$$G_1(j\omega) = 1 + j\omega T$$

$$|G_1(j\omega)| = \sqrt{1 + \omega^2 T^2}$$

$$\angle G_1(j\omega) = \tan^{-1}(\omega T)$$

ω	$\text{Re}(G_1(j\omega))$	$\text{Im}(G_1(j\omega))$	$ G_1(j\omega) $	$\angle G_1(j\omega)$
0	1	0	1	0
$\frac{1}{T}$	1	1	$\sqrt{2}$	$\tan^{-1}(1) = 45^\circ$
∞	1	∞	∞	90°



Eg. $G(s) = \frac{\omega n^2}{s^2 + 2j\omega n s\xi + \omega n^2}$

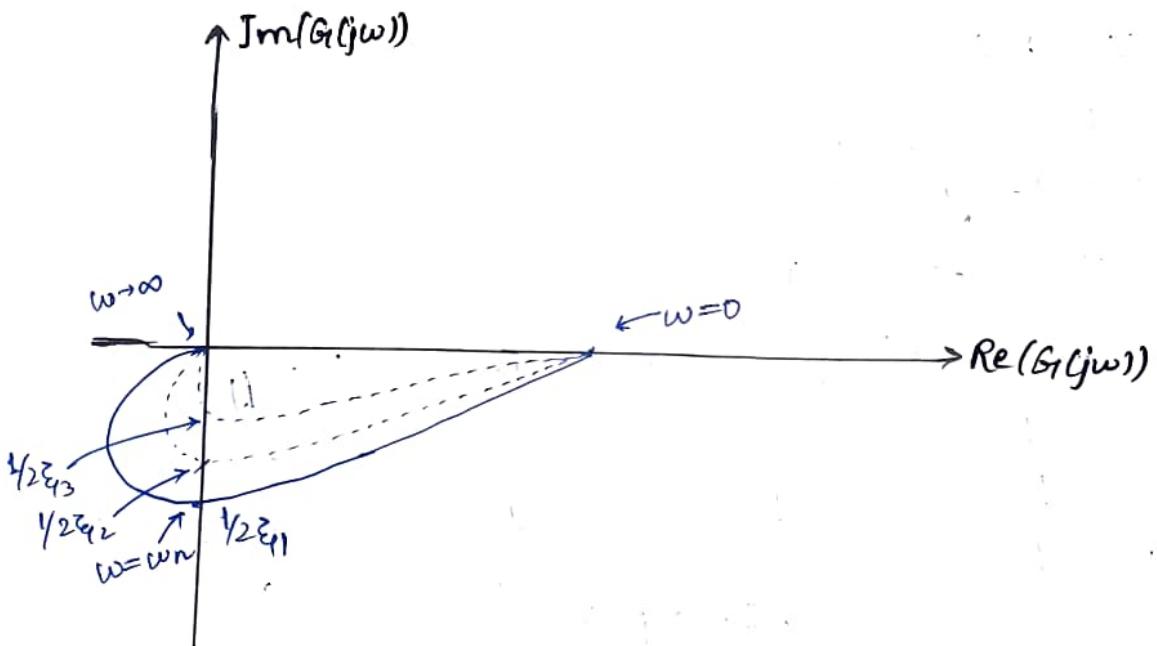
$$G_1(j\omega) = \frac{1}{\left(1 - \frac{\omega^2}{\omega_n^2}\right) + j\left(\frac{2\xi\omega}{\omega_n}\right)} = \frac{1 - \omega^2/\omega_n^2}{\left(1 - \frac{\omega^2}{\omega_n^2}\right)^2 + \left(\frac{2\xi\omega}{\omega_n}\right)^2}$$

$$+ j \sqrt{\frac{-2\xi\omega/\omega_n}{\left(1 - \omega^2/\omega_n^2\right)^2 + \left(2\xi\omega/\omega_n\right)^2}}$$

$$G(j\omega) = \frac{1}{\sqrt{\left(\frac{1-\omega^2}{\omega_n^2}\right)^2 + \left(\frac{2\xi\omega}{\omega_n}\right)^2}} \quad \angle -\tan^{-1}\left(\frac{2\xi\omega/\omega_n}{1-\omega^2/\omega_n^2}\right)$$

$$\Rightarrow G = \frac{1}{a+jb} \quad \angle \tan^{-1}\left(\frac{b}{a}\right)$$

ω	$\operatorname{Re}(G(j\omega))$	$\operatorname{Im}(G(j\omega))$	$ G(j\omega) $	$\angle G(j\omega)$
0	1	0	1	0
ω_n	0	$-1/2\xi$	$1/2\xi$	-90°
∞	0	0	0	-180°



Nyquist Stability

CL char. eqn: $F(s) \triangleq 1 + G_1(s) H(s) = 0$

$$\text{Let } G_1(s) H(s) = \frac{n_o(s)}{d_o(s)} \text{ (OL).}$$

$$\text{Then, } F(s) = 1 + G_1(s) H(s)$$

$$= 1 + \frac{n_o(s)}{d_o(s)} = \frac{d_o(s) + n_o(s)}{d_o(s)}$$

We have (-ve feedback),

$$\frac{Y(s)}{R(s)} = \frac{G_1(s)}{1 + G_1(s) H(s)} = \frac{n'_o(s)/d'_o(s)}{\frac{n_o(s) + d_o(s)}{d_o(s)}}$$

Pole of $F(s) \rightarrow$ OL poles

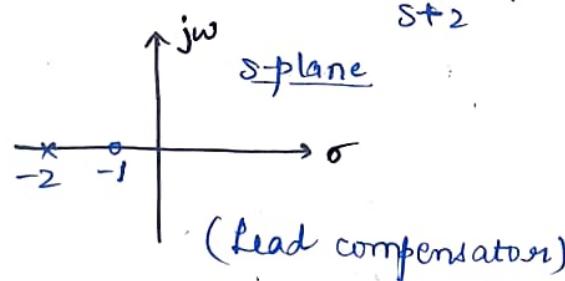
Zeroes of $F(s) \rightarrow$ CL poles

For ~~H(s)~~ $H(s) = 1$,

$$\frac{Y(s)}{R(s)} = \frac{n_o(s)}{n_o(s) + d_o(s)}$$

Mapping Theorem:

Let's consider $G_1(s) = \frac{s+1}{s+2}$

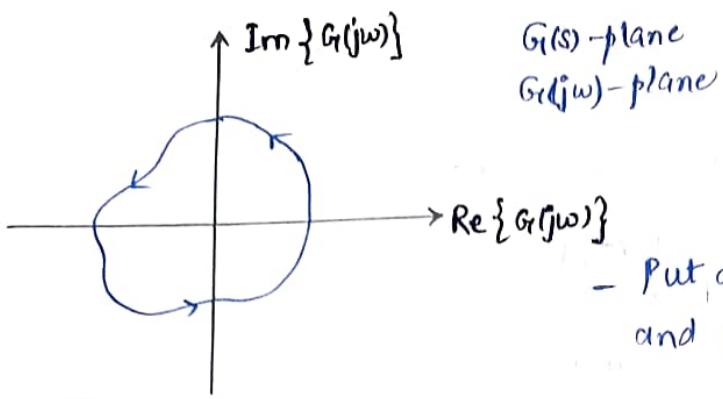


Let $s = -1 + j$ in $G_1(s)$.

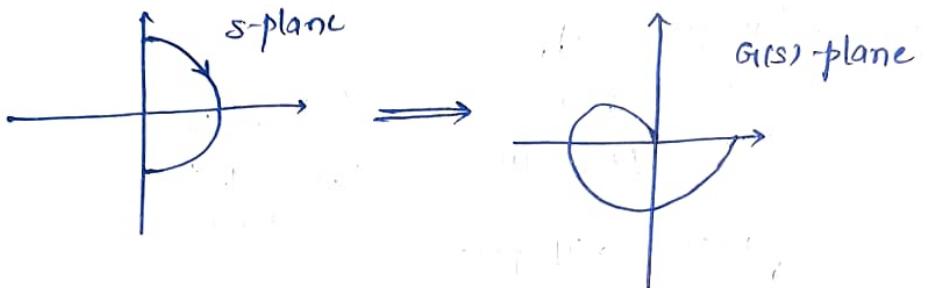
$$G_1(j\omega) = \frac{-1 + j + j}{-1 + j + 2} = \frac{j}{j+1} = \frac{j(1-j)}{2} = \frac{1}{2} + j \frac{1}{2}$$

$$\operatorname{Re}\{G_1(j\omega)\} = \frac{1}{2}$$

$$\operatorname{Im}\{G_1(j\omega)\} = \frac{1}{2}$$



Mapping theorem: Collection of point forms a contour.



Let $F(s)$ be a ratio of two polynomials in s . Let a closed contour in the s -plane contains ' Z ' zeros and ' P ' poles of $F(s)$ without passing over any of the poles and zeros of $F(s)$. Now, this closed contour in the s -plane is mapped into closed contour in $F(s)$ -plane. The total no. of CW encirclements (N_c) of the origin by the contour in $F(s)$ plane is equal to

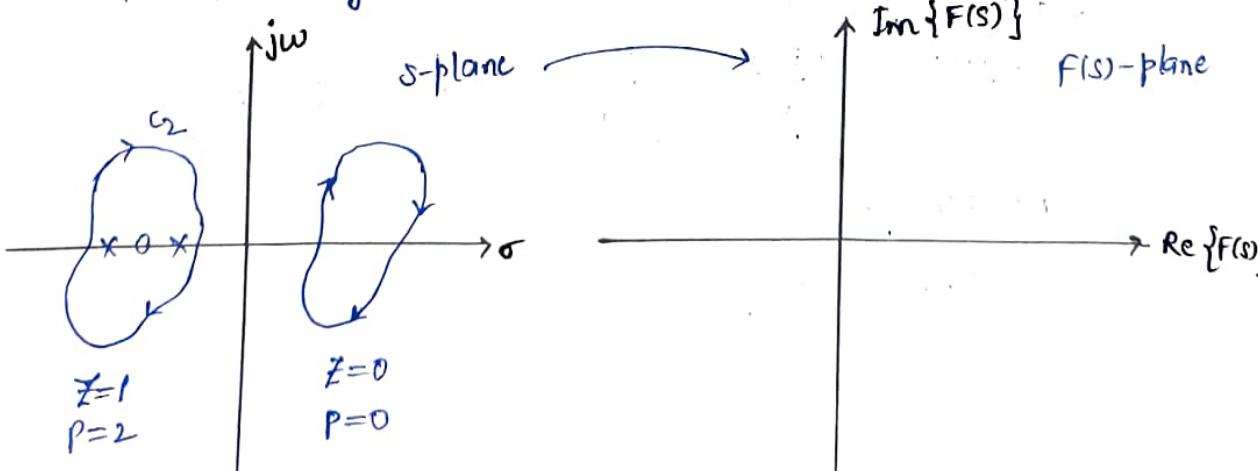
$$N_c = Z - P,$$

Z : No. of zeros of $F(s)$ within closed contour in s -plane.

P : No. of poles of $F(s)$ within closed contour in s -plane.

N_c : No. of CW encirclements of the origin.

For stability, $Z=0 \Rightarrow N_c = -P$

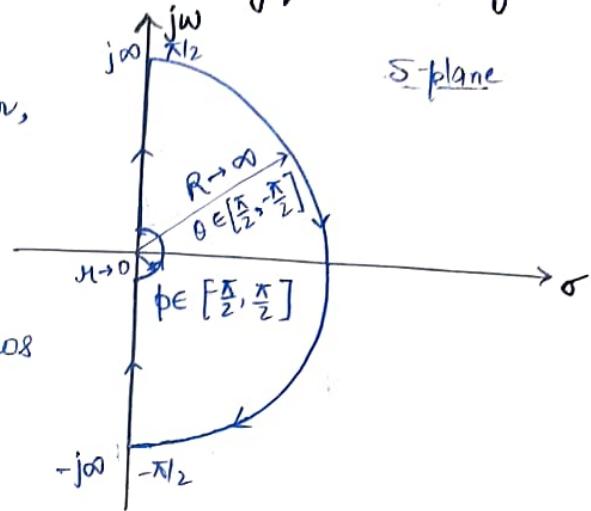


Application of Mapping Theorem to determine Nyquist Stability:

- Circle of infinite radius.
- If there is any pole @ origin, then draw a small circle.

Assumptions:

- i) None of the OL poles and zeros lie in RHP.
- ii) $\lim_{s \rightarrow \infty} G(s)H(s)$ is either 0 or a finite constant.



Nyquist stability criteria: Consider OLTG $G(s)H(s)$ does not have any poles or zeros on img.-axis. If the OLTG $G(s)H(s)$ has 'k' poles in RH s-plane and if $\lim_{s \rightarrow \infty} G(s)H(s)$ is either 0 or a finite non-zero constant, then for stability of CL system, the locus of $G(j\omega)H(j\omega)$ as 'w' varies from $-\infty$ to ∞ must encircle the $-1+j0$ point 'k' times in ACW dirn.

Eg. Consider a system with unity -ve F/B of

$$G(s) = \frac{K}{(s+2)(s^2+2s+2)}$$

Comment about CL stability using nyquist.

Soln: Poles: ~~s = -2, s = -1 + j~~

Since no OL poles in RHP,

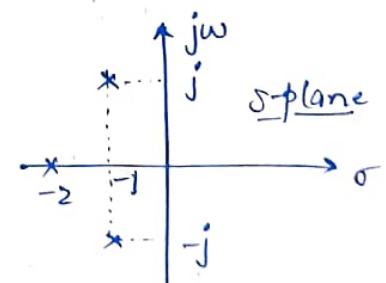
thus nyquist contour should not encircle $-1+j0$ point in ACW dirn.

$$N_c = Z - P = 0 \text{ for stability}$$

$$\text{Let } k=1. \quad G(j\omega) = \frac{1}{(j\omega+2)(-\omega^2+2j\omega+2)}$$

$$\operatorname{Re}\{G(j\omega)\} = \frac{4(1-\omega^2)}{16(1-\omega^2)^2 + \omega^2(6-\omega^2)^2}$$

$$\operatorname{Im}\{G(j\omega)\} = \frac{-\omega^2(6-\omega^2)}{16(1-\omega^2)^2 + \omega^2(6-\omega^2)^2}$$



Set $\operatorname{Re}\{G_1(j\omega)\} = 0$ to find the intersection of the curve with imaginary axis.

$$\frac{4(1-\omega^2)}{16(1-\omega^2)^2 + \omega^2(6-\omega^2)^2} = 0$$

$$16(1-\omega^2)^2 + \omega^2(6-\omega^2)^2$$

$$\Rightarrow \omega^2 = 1 \Rightarrow \omega = \pm 1$$

Put this value in $\operatorname{Im}\{G_1(j\omega)\}$, we have

$$\frac{-\omega(6-\omega^2)}{16(1-\omega^2)^2 + \omega^2(6-\omega^2)^2} = -\frac{1(6-1)}{16(1-1)^2 + 1(6-1)^2} = -\frac{5}{25} = -\frac{1}{5}$$

For $\omega = -1$, $-\frac{\omega(6-\omega^2)}{16(1-\omega^2)^2 + \omega^2(6-\omega^2)^2} = -\frac{(-1)(6-1)}{16(1-1)^2 + 1(6-1)^2} = \frac{5}{25} = 1/5$

For $\omega = +1$, $-\frac{1(6-1)}{16(1-1)^2 + 1(6-1)^2} = -\frac{1 \times 5}{25} = -1/5$

Again set $\operatorname{Im}\{G_1(j\omega)\} = 0$ to find the intersection of the curve with real axis.

~~$\operatorname{Im}\{G_1(j\omega)\} = 0$~~

$$-\frac{\omega(6-\omega^2)}{16(1-\omega^2)^2 + \omega^2(6-\omega^2)^2} = 0$$

$$\Rightarrow \omega(6-\omega^2) = 0$$

$$\Rightarrow \omega = 0 \text{ or } \omega^2 = 6 \Rightarrow \omega = \pm\sqrt{6}$$

Put these values in $\operatorname{Re}\{G_1(j\omega)\}$,

$$\omega = 0 \Rightarrow \frac{4(1-0)^2}{16(1-0)^2} = \frac{4}{16} = \frac{1}{4}$$

$$\omega = \sqrt{6} \Rightarrow \frac{4(1-6)}{16(1-6)^2 + 6(6-6)^2} = \frac{4(-5)}{16 \times 25} = -\frac{1}{20}$$

$$\omega = -\sqrt{6} \Rightarrow \frac{4(1-6)}{16(1-6)^2 + 6(6-6)^2} = -\frac{1}{20}$$

Also put $\omega = 0$ in $\operatorname{Re}\{G_1(j\omega)\}$

$$\Rightarrow \frac{4(1-0)}{16(1-0)^2 + 0} = 1/4$$

At $\omega = 0$,

$$\operatorname{Im}\{G_1(j\omega)\} = -\frac{0(6-0)}{16(1-0)^2 + 0(6-0)^2} = 0$$

\therefore At $\omega=0$, $(\operatorname{Re}\{G_1(j\omega)\}, \operatorname{Im}\{G_1(j\omega)\}) = (1/4, 0)$.

Put $\omega \rightarrow \infty$ in $\operatorname{Re}\{G_1(j\omega)\}$,

$$\lim_{\omega \rightarrow \infty} \frac{4\left(\frac{1}{\omega^6} - \frac{1}{\omega^4}\right)}{16\left(\frac{1}{\omega^3} - \frac{1}{\omega}\right)^2 + \left(\frac{6}{\omega^2} - 1\right)^2} = \frac{4(0-0)}{16(0-0)^2 + (0-1)^2} = 0$$

[Dividing N^s & D^s by ω^6]

Put $\omega \rightarrow \infty$ in $\operatorname{Im}\{G_1(j\omega)\}$,

$$\lim_{\omega \rightarrow \infty} \frac{\left(\frac{6}{\omega^5} - \frac{1}{\omega^3}\right)}{16\left(\frac{1}{\omega^3} - \frac{1}{\omega}\right)^2 + \left(\frac{6}{\omega^2} - 1\right)^2} = 0$$

$$\therefore \omega=0 \Rightarrow \operatorname{Re}\{G_1(j\omega)\} = 1/4, \operatorname{Im}\{G_1(j\omega)\} = 0$$

$$\omega=\pm 1 \Rightarrow \operatorname{Re}\{G_1(j\omega)\} = 0, \operatorname{Im}\{G_1(j\omega)\} = \pm 1/5$$

$$\omega=\pm i \Rightarrow \operatorname{Re}\{G_1(j\omega)\} = 0, \operatorname{Im}\{G_1(j\omega)\} = \pm 1/5$$

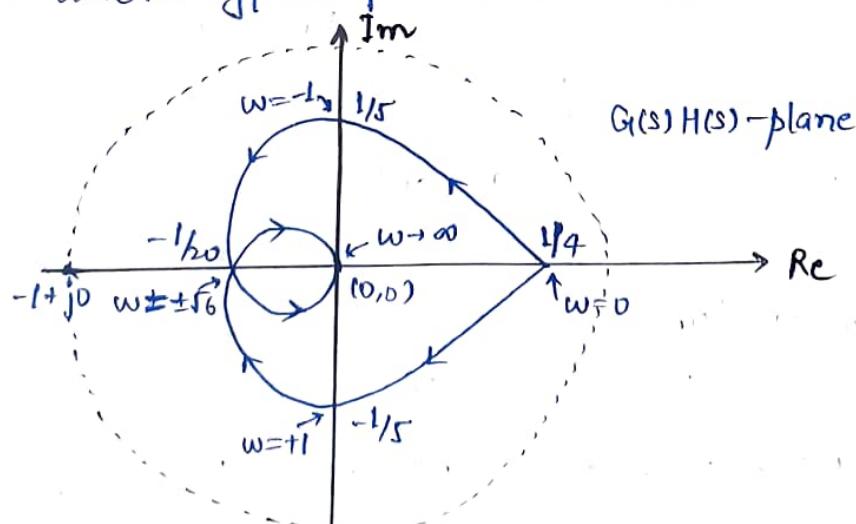
$$\omega=\pm \sqrt{6} \Rightarrow \operatorname{Re}\{G_1(j\omega)\} = -1/20, \operatorname{Im}\{G_1(j\omega)\} = 0$$

$$\omega \rightarrow \infty \Rightarrow \operatorname{Re}\{G_1(j\omega)\} = 0, \operatorname{Im}\{G_1(j\omega)\} = 0.$$

24-11-2025

ω_{gc} : ' ω ' where nyquist plot intersects the unit circle.

ω_{pc} : ' ω ' where nyquist plot intersects the -ve real axis.



$$G_1 M = \frac{1}{|G_1(j\omega)H_1(j\omega)|}_{\omega_{pc}} = \frac{1}{|-1/20|} = 20.$$

$$G_1 M (\text{dB}) = 20 \log_{10} 20 = 26 \text{ dB} > 0$$

$P M \rightarrow \infty$

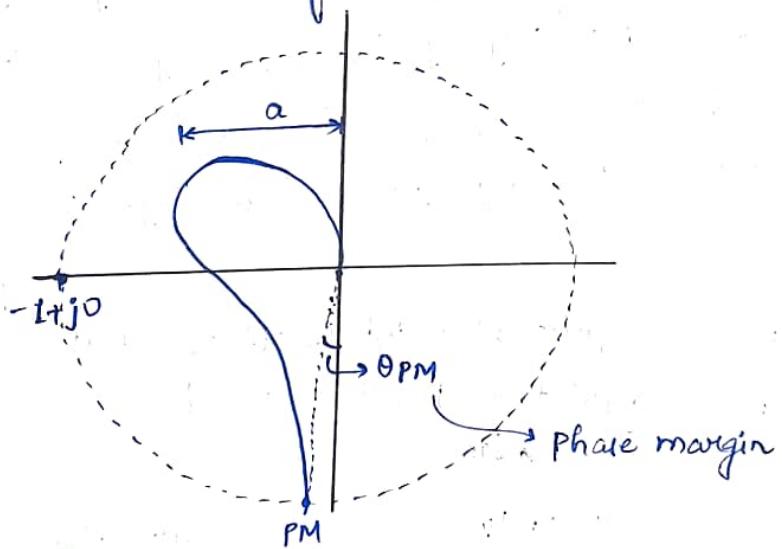
Since there are no poles in RHP, thus ~~the~~ the Nyquist contour should not encircle the $(-1+j0)$ point.

However, for $K=1$, the contour intersects the real axis at $-1/a$ for $\omega = \pm\sqrt{6}$. Thus, for $K < 20$, the contour does not encircle the $(-1+j0)$ point and hence the system is stable for $K < 20$. For $K > 20$, the contour encircles the $(-1+j0)$ point in ACW dir^w and hence the system becomes unstable.

For $K=20$, the system is marginally stable with freq. of oscillation $\omega = \sqrt{6}$ rad/sec.

Root locus : K varies

Nyquist plot: ω varies from 0 to ∞ .



$$GM = \frac{1}{|a|} \rightarrow -20 \log |a| \text{ (dB)}$$

filter / Estimator / Observer:

- Filter and estimator are the same thing.

Kalman Filter: Certain unwanted part rejected.

Estimator \Rightarrow Estimate the o/p (in statistical sense)

$$y = Hx + n$$

\uparrow \uparrow \uparrow
 o/p s/p noise

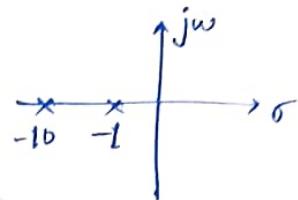
Observer \Rightarrow Deterministic

| # Book by Dorf & Bishop

Eg. Sketch the nyquist plot for the unity f/b control system with $G(s) = \frac{100}{(s+1)(0.1s+1)}$ and comment about stability.

Soln Poles: -1, -10

$$G(j\omega) = \frac{100}{(j\omega+1)(0.1j\omega+1)} \quad [H(s)=1]$$



$$\begin{aligned} G(j\omega) &= \frac{100}{(j\omega+1)(0.1j\omega+1)} \\ &= \frac{100}{-\omega^2 + 1.1j\omega + 1} \\ &= \frac{100}{(1-0.1\omega^2) + 1.1j\omega} \\ &= \frac{100}{(1-0.1\omega^2) + 1.1j\omega} \times \frac{(1-0.1\omega^2) - 1.1j\omega}{(1-0.1\omega^2) - 1.1j\omega} \\ &= \frac{(100 - 10\omega^2) - 110j\omega}{(1-0.1\omega^2)^2 + (1.1\omega)^2} \\ &= \frac{10(10 - \omega^2)}{(1-0.1\omega^2)^2 + (1.1\omega)^2} + j \left(-\frac{110\omega}{(1-0.1\omega^2)^2 + (1.1\omega)^2} \right) \\ &= \frac{10(10 - \omega^2)}{(1+1.01\omega^2) + 0.01\omega^4} + j \left(\frac{110\omega}{(1+1.01\omega^2) + 0.01\omega^4} \right) \end{aligned}$$

$$\therefore \operatorname{Re}\{G(j\omega)\} = \frac{10(10 - \omega^2)}{1+1.01\omega^2+0.01\omega^4}$$

$$\operatorname{Im}\{G(j\omega)\} = -\frac{110\omega}{1+1.01\omega^2+0.01\omega^4}$$

$$\operatorname{Im}\{G(j\omega)\} = 0 \Rightarrow -\frac{110\omega}{1+1.01\omega^2+0.01\omega^4} = 0 \Rightarrow \omega = 0$$

$$\text{Put } \omega = 0 \text{ in } \operatorname{Re}\{G(j\omega)\} \Rightarrow \frac{10(10 - 0)}{1+0+0} = 100$$

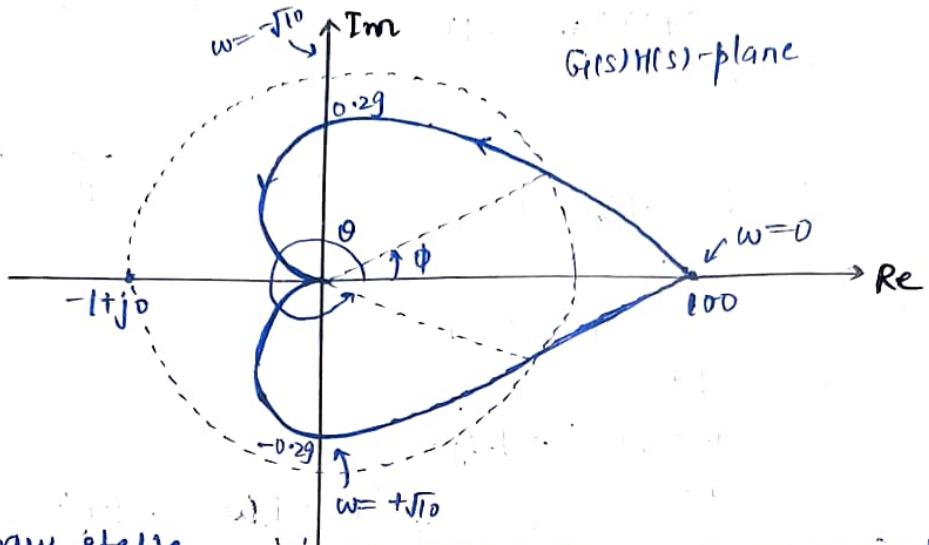
$$\operatorname{Re}\{G(j\omega)\} = 0 \Rightarrow \frac{10(10 - \omega^2)}{1+1.01\omega^2+0.01\omega^4} = 0 \Rightarrow \omega = \pm\sqrt{10}$$

$$\begin{aligned} \text{Put } \omega = \pm\sqrt{10} \text{ in } \operatorname{Im}\{G(j\omega)\} &\Rightarrow -\frac{11(\pm\sqrt{10})}{1+0.01\times 10 + 0.01\times 100} \\ &= -\frac{\pm 3.48}{1+10.1+1} = \mp 0.29 \end{aligned}$$

$$\text{Put } \omega = 0 \text{ in } \operatorname{Im}\{G(j\omega)\} \Rightarrow \operatorname{Im}\{G(j\omega)\}|_{\omega=0} = 0$$

$$\text{Put } \omega \rightarrow \infty \text{ in } \operatorname{Re}\{G(j\omega)\} \Rightarrow \lim_{\omega \rightarrow \infty} \frac{10 \left(\frac{1}{\omega_1} - \frac{1}{\omega_2} \right)}{\frac{1}{\omega_1} + 1 \cdot 0.1 \left(\frac{1}{\omega_2} \right) + 0 \cdot 0.1} = 0$$

$$\text{Put } \omega \rightarrow \infty \text{ in } \operatorname{Im}\{G(j\omega)\} \Rightarrow \lim_{\omega \rightarrow \infty} \frac{-1 \cdot 1 / \omega^2}{\frac{1}{\omega_1} + 1 \cdot 0.1 \left(\frac{1}{\omega_2} \right) + 0 \cdot 0.1} = 0$$



Always stable as '0' DL RHP poles and zero encirclements of $(-1+j0)$ point.

As NP intersects the real axis @ origin, thus

$$GM = 1/0 \rightarrow \infty \Rightarrow GM(\text{dB}) \rightarrow \infty$$

Note: If NP intersects the real-axis at 'a' s.t. $|a| < 1$, then

$$GM = \frac{1}{|a|} \Rightarrow GM = -20 \log |a| > 0.$$

On the other hand, we find ϕ or θ , i.e., the angle made by the origin and the unit circle intersecting point.

$$\text{Thus, } PM = \theta - 180^\circ > 0$$

$$\text{Since, } \Theta = 360^\circ - \phi \Rightarrow -180^\circ + \Theta = 180^\circ - \phi > 0^\circ$$