Assignment 1

AV224 - Control Systems

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Objective: Design of Proportional + Rate feedback, and Proportional + Derivative (PD) type of compensators for a typical electromechanical engine gimbal control system of launch vehicles and linear system performance assessment using SIMULINK model.

1) Find out the values of undamped natural frequency (ωn) , proportional controller gain (K1) and rate feedback controller gain (K2) using a MATLAB program which use the parameter values of the plant dynamics, sensor scale factors and closed specifications. Write down the values below:

MATLAB Code:

Declaring Variables

```
% Motor torque constant
KT = 0.181; \% (N-m/A)
% Motor back emf constant
KB = 0.181; % (V/(rad/sec))
% MI of motor rotating assembly
Jm = 1.1694e-4; \% (Kg-m2)
% MI of Engine gimbal
JL = 12.753; % (Kg-m2)
% Ball screw gear ratio : pitch of ball screw/actuator moment arm x
2pie rad
N = 1/398; % = Nm/NL = 0.005 m/(0.3166 mx 2pie rad)
% Viscous damping coefficient of motor shaft
Bm = 2.943e-4; \% (N-m/(rad/sec))
% Viscous damping coefficient of the Engine gimbal
BL = 58.86; \% (N-m/(rad/sec))
% Armature resistance
Ra = 8.6; % ohms
% Position sensor scale factor
Kp = 0.36; % V/rad (10V corresponds to 4*398 deg deflection of
motor shaft)
% Tacho generator scale factor
KTG = 0.1; % (V/(rad/sec))
% -3 dB Bandwidth
wb = 2*pi*5; \% (rad/2) for fb=5Hz
% damping factor
zeta = 0.6;
```

Parameter Values

```
J = Jm + (N^2)*JL;
B = Bm + KT*KB/Ra + (N^2)*BL;
K = KT/Ra;
wn=wb/(sqrt((1-2*((zeta).^2)) + sqrt(4*((zeta).^4) - 4*((zeta).^2) + 2)))
```

wn = 27.3600

K1 = 19.5077

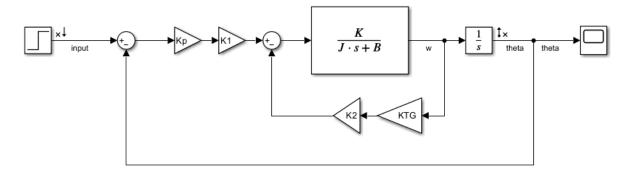
K2 = 0.9538

```
Kd=K2*KTG/Kp;
```

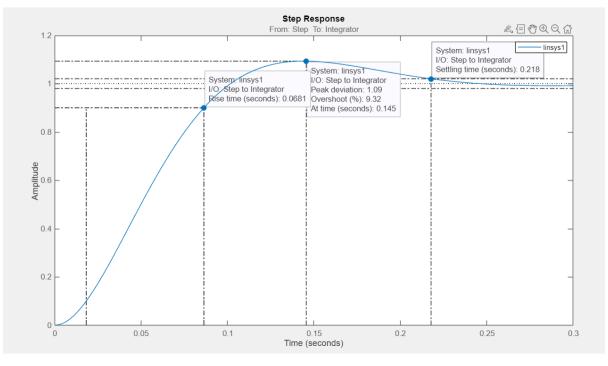
 ω n = 27.36 rad/sec , K1= 19.5077 , K2 = 0.9538

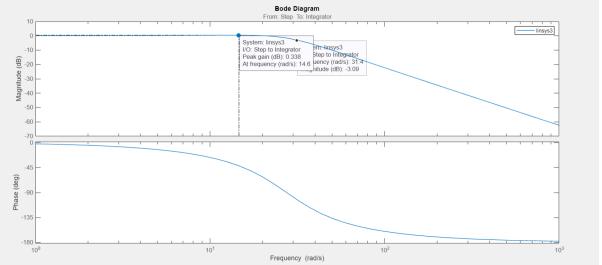
2) Build up a Simulink model for the closed loop system and find out the following performances using SIMULINK LTI Viewer a) – 3 dB bandwidth b) M –peak c) Peak Overshoot for step response d) Rise time e) Settling time

SIMULINK Simulation:



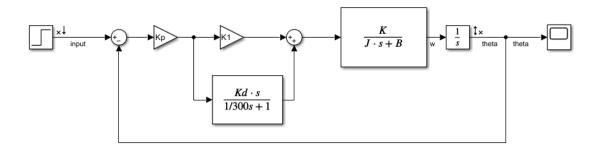
Linear Time Analysis:



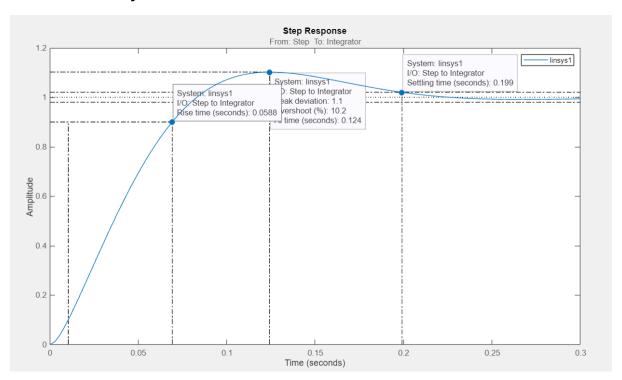


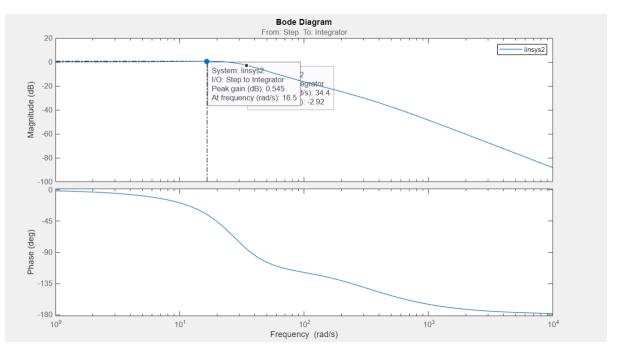
3) Add a derivative controller G(s) = Kd.s/(1+s/300) where Kd = K2*KTG/Kp in parallel with the K1 and remove the rate feedback controller, K2. Repeat the exercise in section (2) for this PD controller.

SIMULINK Simulation:



Linear Time Analysis:





4) Make a performance comparison table as given below:

| Performance | P + Rate feedback | PD Control |
|-------------------|-------------------|------------|
| Parameter | | |
| -3 dB Bandwidth | 4.99 | 5.47 |
| (Hz) | | |
| M peak (dB) | 0.338 | 0.545 |
| Peak Overshoot | 9.32 | 10.2 |
| (%) | | |
| Rise time (m.sec) | 68.1 | 58.8 |
| Settling time | 218 | 199 |
| (m.sec) | | |
| | | |

5) Give your comments on the performance comparison.

Answer:

- (i) In PD controller the speed of response is higher.
- (ii) -3dB Bandwidth: Greater for PD.
- (iii) M peak varies. It's less for PR and higher for PD.
- (iv) %OS is almost the same . It is slightly less for PR controller.
- (v) Rise time is lower for the PD controller.
- (vi) Settling time is lower for PD controller.
- (vii) PD controller performs differently in almost all aspects when compared to PR controller. PD controller reduces SS error at the same time improves transient response. (Lower Ts value).